

# Applied differential equations

## TW244 - Lecture 33

### 10.4 Nonlinear models

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# 10.4: Nonlinear models

## Sliding bead on a wire

Consider the problem of a bead of mass  $m$  sliding along a thin, frictionless wire whose shape is described by the function  $z = f(x)$ .

■ The tangential force is  $mg \sin \theta$

■ In the  $x$  direction:

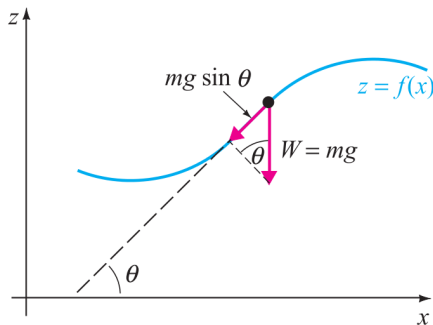
$$\begin{aligned} F_x &= -mg \sin \theta \cos \theta &= -mg \frac{\sin \theta}{\cos \theta} \cos^2 \theta \\ &= -mg \frac{\tan \theta}{\sec^2 \theta} &= -mg \frac{\tan \theta}{1 + \tan^2 \theta} \\ &= -mg \frac{f'(x)}{1 + [f'(x)]^2} \end{aligned}$$

■ Damping force proportional to velocity:

$$D_x = -\beta x'$$

■ Newton's second law ( $ma = F$ ):

$$mx'' = -mg \frac{f'(x)}{1 + [f'(x)]^2} - \beta x' \implies \begin{cases} x' = y \\ y' = -g \frac{f'(x)}{1 + [f'(x)]^2} - \frac{\beta}{m} y \end{cases}$$



# 10.4: Nonlinear models

## Sliding bead on a wire

$$\frac{\partial Q}{\partial x} = -g \frac{f''(x)}{1 + [f'(x)]^2} + \frac{2g[f'(x)]^2 f'(x)}{[1 + [f'(x)]^2]^2}$$

Let's analyse the stability of the system  $x' = y, y' = -g \frac{f'(x)}{1 + [f'(x)]^2} - \frac{\beta}{m} y$ .  
(For now we leave  $f$  general, we will pick an example later.)

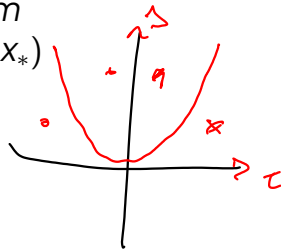
Notice that all critical points require  $y = 0$  and hence  $f'(x) = 0$ . This corresponds to a point where the wire is horizontal - which makes sense!

Consider a critical point  $(x_*, 0)$ , then we have a Jacobian\* where  $f'(x_*) = 0$ .

$$J(x_*, 0) = \begin{bmatrix} 0 & 1 \\ -gf''(x_*) & -\beta/m \end{bmatrix} \Rightarrow \begin{cases} \tau = -\beta/m \\ \Delta = gf''(x_*) \end{cases}$$

and

$$\tau^2 - 4\Delta = (\beta/m)^2 - 4gf''(x_*).$$

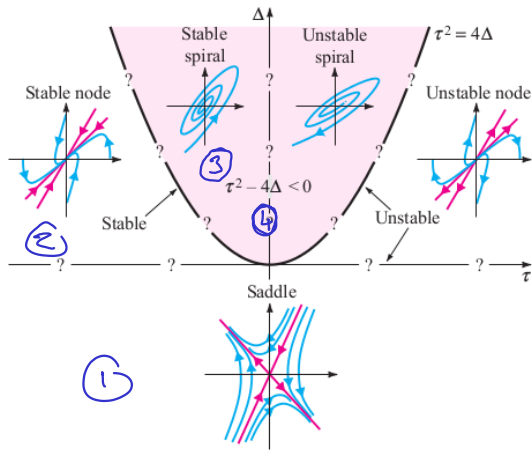


\*Exercise: Show  $\frac{d}{dx} f'(x) / (1 + [f'(x)]^2) \Big|_{x_*} = f''(x_*)$  when  $f'(x_*) = 0$ .

## Sliding bead on a wire

We have  $\tau = -\beta/m$ ,  $\Delta = gf''(x_*)$ ,  $\tau^2 - 4\Delta = (\beta/m)^2 - 4gf''(x_*)$ .

- $\mathbf{f}''(\mathbf{x}_*) < 0$   
 $\implies \Delta < 0 \rightarrow$  saddle  
 (A local maximum in the wire)
- $0 < \mathbf{f}''(\mathbf{x}_*) < \beta^2/(4gm^2)$   
 $\implies \tau^2 - 4\Delta > 0 \rightarrow$  stable node  
 (overdamped)
- $0 < \beta^2/(4gm^2) < \mathbf{f}''(\mathbf{x}_*)$   
 $\implies \tau^2 - 4\Delta < 0 \rightarrow$  stable spiral  
 (underdamped)
- $0 = \beta < \mathbf{f}''(\mathbf{x}_*)$   
 $\implies \tau^2 - 4\Delta < 0 \rightarrow$  centre\*  
 (no damping!)



\* Actually this is borderline and uncertain, but it can be shown by the phase-plane method to be a centre.

# 10.4: Nonlinear models

## Sliding bead on a wire

**Example:** Let's consider an example where the wire is described by the function  $z = f(x) = \sin(x)$ . Let's assume the mass of the bead is  $m = 0.01$ , and (to keep the numbers nicer)  $g = 10$ .

The motion of the bead is then described by the autonomous system

$$x' = y, \quad y' = -10 \cos(x)/(1 + \cos^2(x)) - 100\beta y.$$

The critical points of this system occur when  $f'(x) = \cos(x) = 0$  and  $y = 0$ , i.e.,  $(x, y) = ([k + \frac{1}{2}]\pi, 0)$ ,  $k \in \mathbb{Z}$ .

Rather than rederive the the nature of these critical points from scratch, let's use the general formula from the previous slide.

When  $k$  is odd then  $f''([k + \frac{1}{2}]\pi) = -1$  and so the critical point is a saddle.

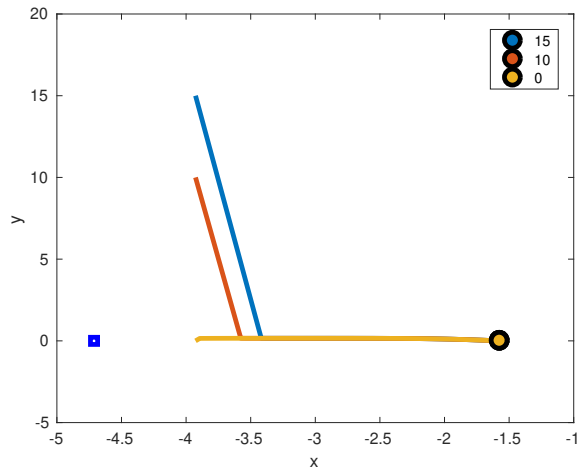
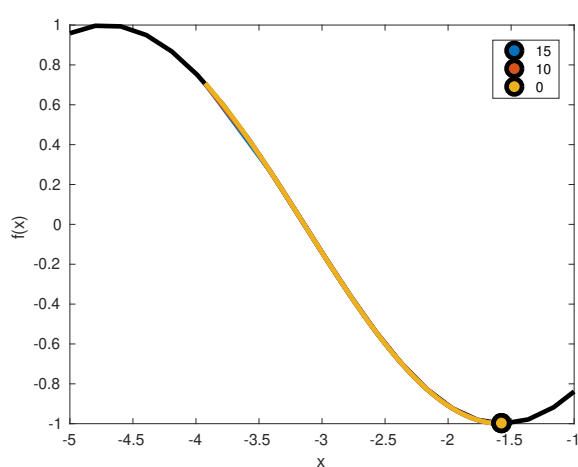
When  $k$  is even the critical value of  $\beta \neq 0$  is when  $0 < f''([k + \frac{1}{2}]\pi) = \beta^2/(4gm^2)$ , i.e.,  $\beta^* = 0.0632$ . Let's consider three cases...

# 10.4: Nonlinear models

## Sliding bead on a wire

### Case 1:

$\beta = 0.07 > \beta^* \implies$  stable point  $\implies$  overdamped.

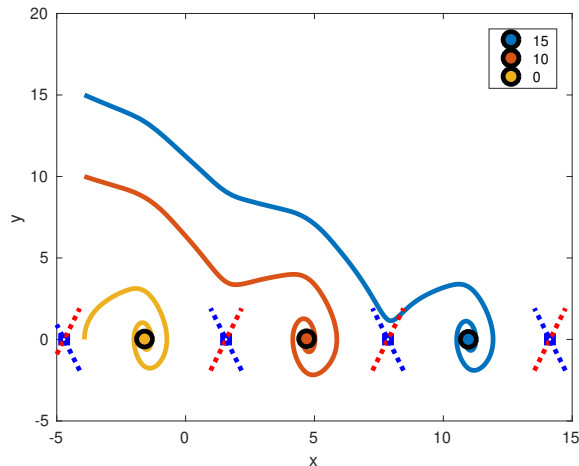
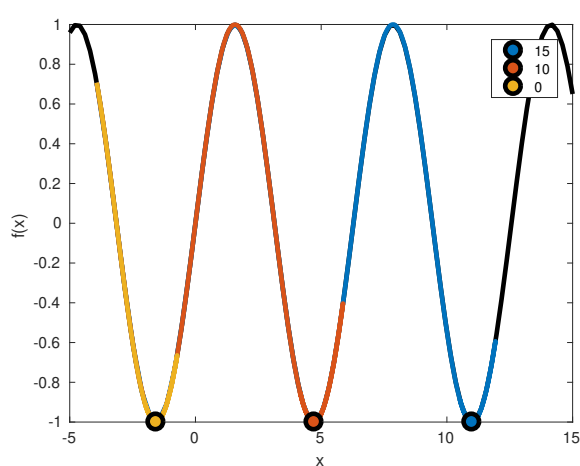


# 10.4: Nonlinear models

## Sliding bead on a wire

### Case 2:

$\beta = 0.01 < \beta^* \implies$  stable spiral  $\implies$  underdamped.

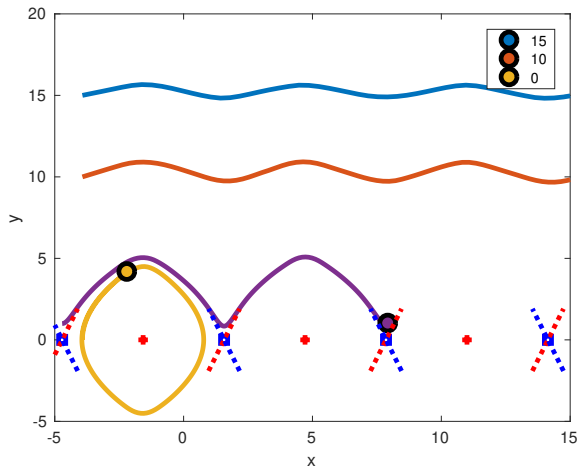
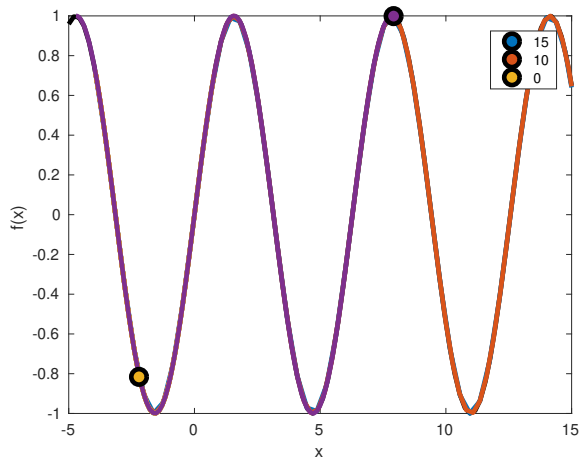


# 10.4: Nonlinear models

## Sliding bead on a wire

### Case 3:

$\beta = 0 \implies$  no damping  $\implies$  periodic solutions.



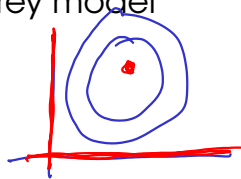


## 10.4: Nonlinear models

### Predator-Prey and competition models

**Lotka–Volterra Predator-Prey model:** Recall the predator-prey model

$$\frac{dx}{dt} = x(-a + by) \quad \frac{dy}{dt} = y(c - dx).$$



It is clear that the two critical points are  $(0,0)$  and  $(\frac{c}{d}, \frac{a}{b})$ .

**Exercise:** Show that  $(0,0)$  is a saddle and that  $(\frac{c}{d}, \frac{a}{b})$  is a borderline case.

One **can** actually use the Phase Plane method to show that  $(\frac{c}{d}, \frac{a}{b})$  is **always** a centre, but the proof is a little tricky, and we omit it here.

**Self-study: Lotka–Volterra competition models:** Incorporating logistic growth in the model above or considering ‘competition models’ (see p. 419 of Z&W) can introduce more complex behaviour, and phase diagrams are an excellent means of analysing this. See p. 419-420 in Z&W and/or play around with some models using the `ppplane` software.

