# Applied differential equations

TW244 - Lecture 32

10.4 Nonlinear models

Prof Nick Hale - 2020





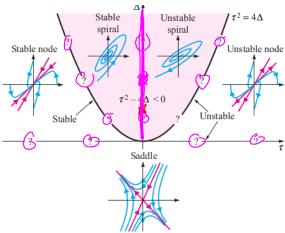
To sketch phase plane of nonlinear planer autonomous DES! (D) Locate catrical points (E) Compute Sacobian metros,  $S = \begin{bmatrix} 38 & 39 \\ 39 & 39 \end{bmatrix}$ 3 Claserfy critical points by substituting to I and diecking (I, A) diagram - see next page. 5 sketch edutions at critical points. (using evels end evers) (5) Sketch solutions between entirel points

(using null dines to help)

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**Important to note:** For nonlinear systems the method of linearisation cannot concretely classify criticial solutions in the borderline cases! Compare the figure above with the one we saw in Lecture 30.



**Example:** Consider the nonlinear spring:

$$x'' = -2x - 3x^3.$$

We may rewrite this as

$$\frac{dx}{dt} = y$$
,  $\frac{dy}{dt} = -x(2+3x^2)$ .

We see immediately that the only critical point is at (0,0) and

$$J(x,y) = \begin{bmatrix} 0 & 1 \\ -2 - 9x^2 & 0 \end{bmatrix} \implies J(0,0) = \begin{bmatrix} 0 & 1 \\ -2 & 0 \end{bmatrix} \implies \Delta = 2, \tau = 0.$$

This is a borderline case and the solution may be a stable spiral, an unstable spiral, or a centre! How can we decide?

The phase-plane method is based on the fact that  $\frac{dy}{dx} = \frac{dy/dt}{dx/dt} = \frac{Q(x,y)}{P(x,y)}$ . Solving for y(x) sometimes tells us the nature of borderline critical points. Returning to our example, we find

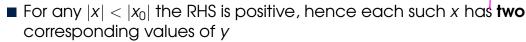
$$\frac{dy}{dx} = -\frac{x(2+3x^2)}{y} \quad \stackrel{\text{tutorial 7}}{\Longrightarrow} \quad y^2 = -x^2 \frac{3}{2}(x^2 + \frac{4}{3}) + c^2$$

Choosing an initial condition  $(x_0,0)$  we find  $c^2 = \frac{3}{2}x_0^2(x_0^2 + \frac{4}{3})$  and therefore\*

$$y^2 = \frac{3}{2}(\frac{4}{3} + x_0^2 + x^2)(x_0^2 - x^2)$$

Noting that

$$y(-x_0) = y(x_0) = 0$$



it follows the solution forms a closed curve and (0,0) is therefore a **centre**.

<sup>\*</sup>complete the square

$$y^{2} = -x^{2} \frac{3}{2} (x^{2} + \frac{4}{3}) + c^{2}, \quad c^{2} = \frac{3}{2} x_{0}^{2} (x_{0}^{2} + \frac{4}{3})$$

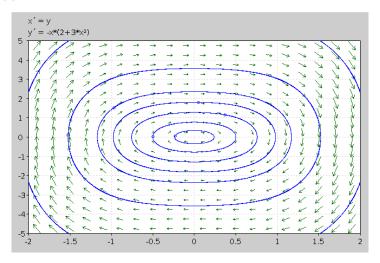
$$y^{2} = -x^{2} \frac{3}{2} (x^{2} + \frac{4}{3}) + x^{3} (x_{0}^{2} + \frac{4}{3})$$

$$= \frac{3}{2} \left[ x_{0}^{2} (x_{0}^{2} + \frac{4}{3}) - x^{2} (x^{2} + \frac{4}{3}) \right] = \frac{3}{2} \left[ x_{0}^{2} (x_{0}^{2} + \frac{4}{3}) - x^{2} (x^{2} + \frac{4}{3}) \right]$$

$$= \frac{3}{2} \left[ x_{0}^{2} (x_{0}^{2} + \frac{4}{3}) - x^{2} (x^{2} + \frac{4}{3}) \right]$$

$$= \frac{3}{2} \left[ x_{0}^{2} (x_{0}^{2} + \frac{4}{3}) - x^{2} (x^{2} + \frac{4}{3}) \right]$$

#### Confirm with pplane:



For more examples of using the Phase Plane method see p. 411-412 in Z&W.

### **NONLINEAR PENDULUM**

## 10.4: Nonlinear models Nonlinear pendulum

Recall the nonlinear pendulum from Lecture 26:

$$\frac{d^2\theta}{dt^2} = -\omega^2 \sin\theta \quad : \quad \omega^2 = \frac{g}{\ell}$$

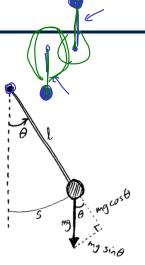
Let  $x = \theta$  and  $y = \frac{d\theta}{dt}$  then we may model the nonlinear pendulum as the plane autonomous system

$$\frac{\frac{dx}{dt} = y,}{\frac{dy}{dt} = -\omega^2 \sin x.}$$

Critical solutions: 
$$\frac{dx}{dt} = \frac{dy}{dt} = 0$$

$$\implies y = 0 \& \sin(x) = 0 \implies x = 0, \pm \pi, \pm 2\pi, \ldots \implies (x, y) = (k\pi, 0) \forall k \in \mathbb{Z}.$$

(1) What positions do these correspond to? (2) Can we classify them?



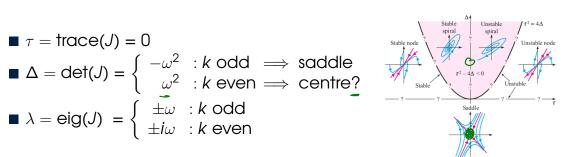
#### 10.4: Nonlinear models Nonlinear pendulum

The system is nonlinear, so we need the Jacobian:

$$J(x,y)=\left[egin{array}{cc} 0 & 1 \ -\omega^2\cos(x) & 0 \end{array}
ight]$$
 and we have  $J(k\pi,0)=\left[egin{array}{cc} 0 & 1 \ \omega^2(-1)^{k+1} & 0 \end{array}
ight]$  with

$$\blacksquare \tau = \mathsf{trace}(J) = 0$$

$$\Delta = \det(J) = \begin{cases} -\omega^2 : k \text{ odd} \implies \text{saddle} \\ \omega^2 : k \text{ even} \implies \text{centre?} \end{cases}$$



When k is odd we get saddle solutions, but k even gives borderline cases.

Let's use the phase-plane method to determine if  $(\pm 2k\pi, 0)$  are centres.

### 10.4: Nonlinear models Nonlinear pendulum

The Costal

Recall: The phase-plane method attempts to relate x and y by dividing

$$\frac{dy}{dt}$$
 by  $\frac{dx}{dt}$  and eliminating  $t$ . In this case

$$\frac{dy}{dx} = \frac{dy/dt}{dx/dt} = -\omega^2 \frac{\sin(x)}{y}.$$

We can solve via separation of variables to find  $y^2 = 2\omega^2 \cos(x) + c^{-1}$  and if we choose an initial point on the x axis, i.e.,  $(x_0, 0)$ , then we have

$$y^2 = 2\omega^2[\cos(x) - \cos(x_0)].$$

Similar to nonlinear spring in the previous lecture, we can now show that

$$y(-x_0) = y(x_0) = 0$$

■ For any  $|x| < |x_0| < \pi$  the RHS is positive, hence each such x has **two** corresponding values of y

it follows the solution forms a closed curve and (0,0) is therefore a **centre**.

We can apply this same idea to each critical point  $(\pm 2k\pi, 0), k = 1, 2, ...$ 

# 10.4: Nonlinear models Nonlinear pendulum

We are almost ready to sketch our phase diagram, but let's first find the direction of the saddles at  $(\pm [2k+1]\pi,0)$ . The eigenvalues of  $+\omega$  and  $-\omega$  have corresponding eigenvectors  $[1,\omega]^{\top}$  and  $[1,-\omega]$ , respectively. The figure on the right therefore shows the behaviour of solutions in the neighbourhood of these critical points.

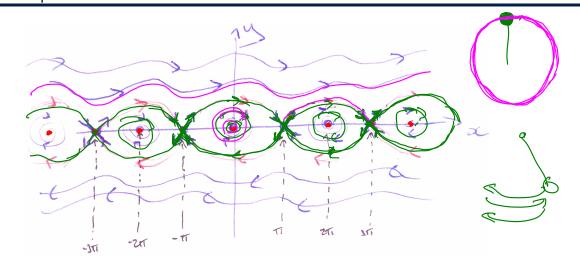
Recalling that for initial condition  $(x_0,0)$  we had  $y^2=2\omega^2[\cos(x)-\cos(x_0)]$ . We can substitute  $x_0=\pi$  to find the equation

$$y^2 = 2\omega^2(\cos(x) - (-1)) = 4\omega^2\cos^2(x/2) \implies y = \pm 2\omega\cos(x/2)$$

which connects each of the saddle points.

Let's combine this with the centres at  $(k\pi,0)$  to sketch the full figure.

### 10.4: Nonlinear models Nonlinear pendulum



We can observe two kinds of motion, determined by the initial conditions.

periodic oscillation



whirling about the pivot



# 10.4: Nonlinear models Nonlinear pendulum

