Applied differential equations

TW244 - Lecture 24

5.1: Spring-mass systems (cont.)

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5.1: Spring-mass systems (cont.) Driven motion without damping: Recall

The undamped forced system
$$x'' + \omega^2 x = F_0 \cos(\gamma t), \quad x(0) = x'(0) = 0, \quad 0 < \gamma \neq \omega$$

has solution

$$X(t) = \frac{F_0}{\omega^2 - \gamma^2} (\cos(\gamma t) - \cos(\omega t)).$$

We require $\gamma \neq \omega$, but what if

- $\gamma \approx \omega ?$
- $\blacksquare \gamma \rightarrow \omega$?

5.1: Spring-mass systems (cont.) x()= To (cost E-cout)

Let's first write x in a different form. Remember that = $\frac{2}{2}$ $\frac{1}{2}$ $\frac{1$

$$\cos(\theta + \phi) = \cos\theta\cos\phi - \sin\theta\sin\phi$$

$$\cos(\theta - \phi) = \cos\theta\cos\phi + \sin\theta\sin\phi$$

$$\therefore\cos(\theta - \phi) - \cos(\theta + \phi) = 2\sin\theta\sin\phi$$

In our case,

$$\left. \begin{array}{l}
\theta - \phi = \gamma t \\
\theta + \phi = \omega t
\end{array} \right\} \implies \left. \begin{array}{l}
\theta = \frac{1}{2}(\omega + \gamma)t \\
\phi = \frac{1}{2}(\omega - \gamma)t
\end{array} \right]$$

Hence

$$x(t) = \frac{2F_0}{\omega^2 - \gamma^2} \sin\left[\frac{1}{2}(\omega + \gamma)t\right] \sin\left[\frac{1}{2}(\omega - \gamma)t\right].$$

Case 1: Let's consider $\left| \gamma = \omega + 2\varepsilon \right|$ where $\varepsilon \ll$ 1, then we have that

$$\frac{1}{2}(\omega + \gamma) = \frac{1}{2}(\gamma - 2\varepsilon + \gamma) = \gamma - \varepsilon \approx \gamma$$

$$\frac{1}{2}(\omega - \gamma) = \frac{1}{2}(\gamma - 2\varepsilon - \gamma) = -\varepsilon$$

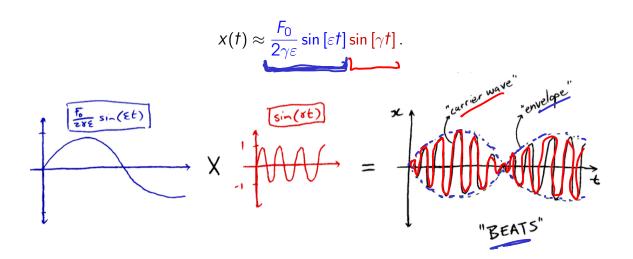
$$(\omega^2 - \gamma^2) = (\omega + \gamma)(\omega - \gamma) \approx -4\gamma\varepsilon$$

Substituting these approximations to

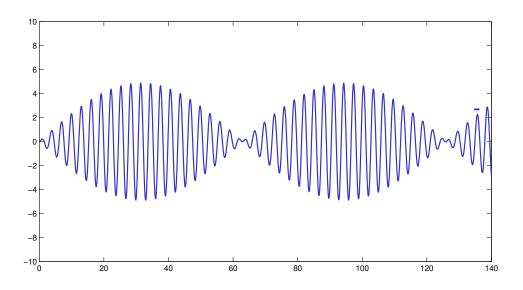
gives
$$x(t) = \frac{2F_0}{\omega^2 - \gamma^2} \sin\left[\frac{1}{2}(\omega + \gamma)t\right] \sin\left[\frac{1}{2}(\omega - \gamma)t\right]$$
$$= \underbrace{2F_0}_{\text{VI}} \sin\left[\frac{1}{2}(\omega + \gamma)t\right] \sin\left[\frac{1}{2}(\omega - \gamma)t\right]}_{\text{X}(t) \approx \frac{F_0}{2\gamma\varepsilon}} \sin\left[\varepsilon t\right] \sin\left[\gamma t\right].$$

What would the graph of such a function look like?

5.1: Spring-mass systems (cont.) Driven motion without damping



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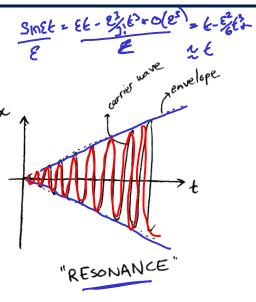
Case 2: Let's now consider the case $\gamma \to \omega$. Equivalently, let $\varepsilon \to 0$ then we have that

$$x(t) = \lim_{\varepsilon \to 0} \frac{F_0}{2\gamma} \frac{\sin(\varepsilon t)}{\varepsilon} \sin(\gamma t)$$
$$= \frac{F_0 t}{2\omega} \sin(\omega t)$$

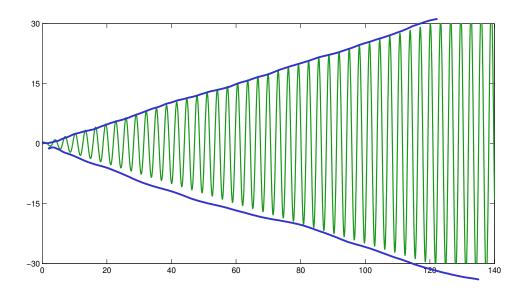
i.e., a sine wave with pseudo-amplitude $\frac{t_0}{2}t$, which **arows** linearly with time!

We call this "pure resonance" (as there is no damping present, which is unphysical).

See CAO4 for damped resonance.



5.1: Spring-mass systems (cont.) Driven motion without damping



5.14: Series circuit analogue Self-study

Many different physical systems can be described by a linear second order differential equation similar to the differential equation of forced motion with damping.

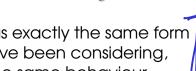
For example, if i(t) denotes current in the LRC-series electrical circuit shown in the Figure, then, by Kirchhoff's second law, the sum of the voltages drops equals the voltage E(t) impressed on the circuit; that is,

$$L\frac{di}{dt} + Ri + \frac{1}{C}q = E(t).$$

But $i = \frac{dq}{dt}$, so

$$L\frac{d^2q}{dt^2} + R\frac{dq}{dt} + \frac{1}{C}q = E(t).$$

Although the letters are different, this equation has exactly the same form as the forced-damped spring-mass system we have been considering, and we can conclude that such circuits exhibit the same behaviour.



5.3: NONLINEAR MODELS

5.3: Nonlinear models

Example 1: Nonlinear extensions of the spring-mass system

Example 1a: Nonlinear spring

Assume a spring force of the form $F = k_1 x + k_2 x^3$

$$\implies m\frac{d^2x}{dt^2} = -(k_1x + k_2x^3)$$

$$\implies x'' + \omega_1x + \omega_2x^3 = 0.$$

This is known as 'Duffing's equation'. We cannot solve it analytically.)=

Example 1b: Quadratic damping

Assume the magnitude of the damping force is proportional to the square of the velocity (and direction opposite to motion)

$$\implies m \frac{d^2x}{dt^2} = -kx - \beta \left| \frac{dx}{dt} \right| \frac{dx}{dt}.$$

(Note that $\left(\frac{dx}{dt}\right)^2$ won't work. $\left|\frac{dx}{dt}\right|\frac{dx}{dt}$ incorporates the direction of motion.)

This DE is also not analytically solveable.)=

5.3: Nonlinear models Example 2: Nonlinear pendulum

Example 2: Nonlinear pendulum

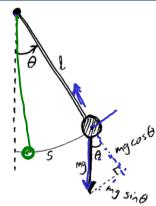
Let $\theta(t)$ be the (signed) angle that the pendulum forms with the vertical and s(t) be the arc length.

Tangential force:
$$F_t = -mg \sin \theta = m \frac{d^2s}{dt^2}$$

But $s = \ell\theta \implies \frac{d^2s}{dt^2} = \ell \frac{d^2\theta}{dt^2} \implies m\ell \frac{d^2\theta}{dt^2} = -mg\sin\theta$ i.e.,

 $\frac{d^2\theta}{dt} + \omega^2 \sin\theta = 0$

$$\frac{d^2\theta}{dt^2} + \omega^2 \sin \theta = 0 \qquad : \quad \omega^2 = \frac{g}{l}.$$



This DE can be solved 'analytically': $\theta(t) = 2\arcsin(\sqrt{m}\operatorname{sn}(K(m) - \omega t; m))$ where $m = \sin^2(\theta_0/2)$, $\operatorname{sn}(\cdot; m)$ is a Jacobi elliptic function (ellipti(.,m) in MATLAB), and K(m) is the complete elliptic integral (ellipke (m)).

See here or here for a partial derivation (not examinable). Note $m = k^2$.

For $\theta \ll 1$ the approximation $\sin \theta \approx \theta$ gives the linear pendulum $\frac{d^2\theta}{d\theta^2} + \omega^2\theta = 0$.