

Applied differential equations

TW244 - Lecture 22

5.1: Spring-mass systems (cont.)

Prof Nick Hale - 2020



SCIENCE
NATUURWETENSAPPE
EYOBUNZULULWAZI

Spring-mass systems (cont.)

Damped motion

Damped motion:

Suppose now that there is also a linear damping force in the direction opposite to motion (e.g., due to air resistance or friction):

$$m \frac{d^2x}{dt^2} = -kx - \beta \frac{dx}{dt}.*$$

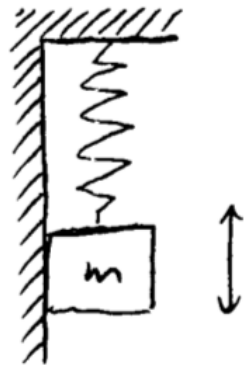
Therefore

$$x'' + 2\gamma x' + \omega^2 x = 0 \quad \text{with} \quad \omega^2 = \frac{k}{m}, \quad 2\gamma = \frac{\beta}{m}.$$

This is a linear homogeneous DE!

Try $x = e^{pt}$ as a solution (we use p here as m is already used for the mass).

* Convince yourself that the term in red is damping regardless of whether the spring moves up or down. 🙋



Spring-mass systems (cont.)

Damped motion (reminder)

Recall that we derived an equation for damped motion as

$$x'' + 2\gamma x' + \omega^2 x = 0,$$

where $2\gamma = \beta/m$ and $\omega^2 = k/m$.^{*} Substituting $x = e^{pt}$ gives

$$\underline{p^2 + 2\gamma p + \omega^2 = 0},$$

$$x = e^{pt}$$

and therefore

$$\underline{p = -\gamma \pm \sqrt{\gamma^2 - \omega^2}}.$$

This leads to three cases:

- $\gamma^2 > \omega^2 \implies$ two real roots (“overdamped”)
- $\gamma^2 = \omega^2 \implies$ one real root (“critically overdamped”)
- $\gamma^2 < \omega^2 \implies$ no real roots (“underdamped”)

^{*}Recall that β is the **damping** constant, k is the **spring** constant, and m is the mass.

Spring-mass systems (cont.)

Damped motion (cont.)

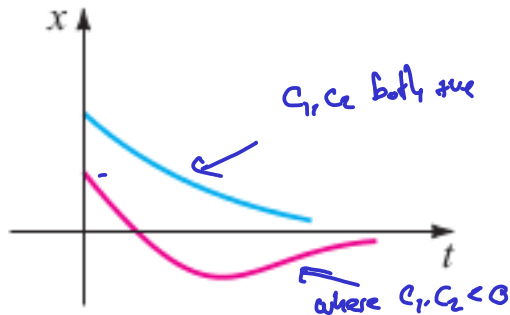
Case 1: $\gamma^2 > \omega^2 \Rightarrow$ two real roots ("overdamped")

$$\gamma^2 > \omega^2 \Rightarrow \frac{\beta}{4m^2} > \frac{k}{m} \Rightarrow \underline{\underline{\beta^2 > 4km.}}$$



Then $p_1 = -\gamma + \sqrt{\gamma^2 - \omega^2} < 0$ and $p_2 = -\gamma - \sqrt{\gamma^2 - \omega^2} < 0$ and

$$\begin{aligned} x(t) &= c_1 e^{p_1 t} + c_2 e^{p_2 t} \quad \leftarrow \\ &= e^{-\gamma t} \left(c_1 e^{+\sqrt{\gamma^2 - \omega^2} t} + c_2 e^{-\sqrt{\gamma^2 - \omega^2} t} \right) \\ &= \underline{\underline{c_1 e^{p_2 t} \left(e^{2\sqrt{\gamma^2 - \omega^2} t} + \frac{c_2}{c_1} \right)}} \end{aligned}$$



Note that $x \rightarrow 0$ as $t \rightarrow \infty$,
i.e., it returns to the equilibrium position.

Notice there is at most one 'oscillation' (if c_1 and c_2 are of different signs).

Spring-mass systems (cont.)

Damped motion (cont.)

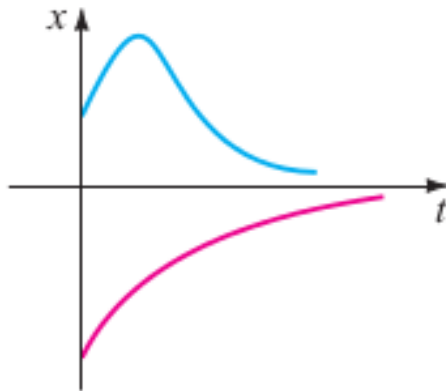
Case 2: $\gamma^2 = \omega^2$ \implies one real root ("critically damped")

$$\gamma^2 = \omega^2 \implies \underline{\beta^2 = 4km} \quad \text{and} \quad \underline{p = -\gamma}.$$

Then we have that (recall Lecture 15)

$$\begin{aligned} x(t) &= c_1 e^{pt} + c_2 t e^{pt} \\ &= \underline{c_2 e^{-\gamma t}} \left(\underline{t + \frac{c_1}{c_2}} \right) \end{aligned}$$

and again note that $x \rightarrow 0$ as $t \rightarrow \infty$
and that there is at most one oscillation.



Spring-mass systems (cont.)

Damped motion (cont.)

Case 3: $\gamma^2 < \omega^2 \implies$ no real roots ("underdamped")

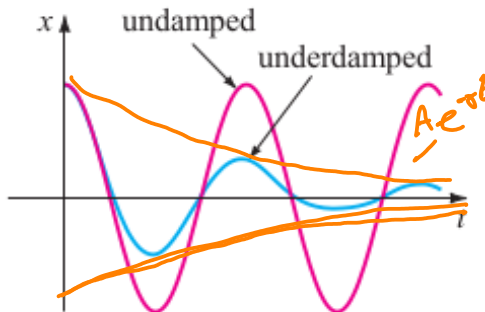
$$\gamma^2 < \omega^2 \implies \frac{\beta}{4m^2} > \frac{k}{m} \implies \beta^2 < 4km.$$

Then $p_1 = -\gamma + i\sqrt{\omega^2 - \gamma^2} < 0$ and $p_2 = -\gamma - i\sqrt{\omega^2 - \gamma^2} < 0$ and[†]

$$\begin{aligned} x(t) &= c_1 e^{p_1 t} + c_2 e^{p_2 t} \\ &= e^{-\gamma t} \left(d_1 \cos(\sqrt{\omega^2 - \gamma^2} t) + d_2 \sin(\sqrt{\omega^2 - \gamma^2} t) \right) \\ &= \underbrace{Ae^{-\gamma t}} \sin(\sqrt{\omega^2 - \gamma^2} t + \phi) \end{aligned}$$

Note that the "amplitude" decays exponentially.

We call this a "damped oscillator".



[†]Recall Lecture 15.

Spring-mass systems (cont.)

Example 1

$$x'' + 2\gamma x' + \omega^2 x = 0$$

Example 1: Consider the spring-mass system described by

$$\underline{x'' + 4x' + 4x = 0} \quad \text{with} \quad \underline{x(0) = 1} \quad \text{and} \quad \underline{x'(0) = -2}.$$

(Note $\omega^2 = 4$ and $\gamma = 2$.) If we try $\underline{x = e^{pt}}$ we have

$$\underline{p^2 + 4p + 4 = (p + 2)^2 = 0} \implies \underline{p = -2},$$

i.e., one real root \implies critical damping and therefore

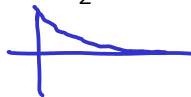
$$x = c_1 e^{-2t} + c_2 t e^{-2t} \quad \leftarrow$$

$$x' = -2c_1 e^{-2t} + c_2(1 - 2t)e^{-2t}.$$

$$\text{Initial conditions } \begin{cases} x(0) = 1 \implies 1 = c_1 + 0 \\ x'(0) = -2 \implies -2 = -2c_1 + c_2 \end{cases} \implies \begin{cases} c_1 = 1 \\ c_2 = 0 \end{cases}.$$

Hence,

$$\boxed{x(t) = e^{-2t}}.$$



Spring-mass systems (cont.)

Example 2

Example 2: Consider the spring-mass system described by

$$x'' + 2x' + 4x = 0 \quad \text{with} \quad x(0) = 1 \quad \text{and} \quad x'(0) = -2.$$

If we try $x = e^{pt}$ we have

$$p^2 + 2p + 4 = 0 \implies p = -\frac{1}{2}(-2 \pm \sqrt{4 - 16}) = -1 \pm i\sqrt{3}$$

i.e., no real roots \implies underdamped and therefore

$$\begin{aligned} x &= c_1 e^{(-1+i\sqrt{3})t} + c_2 e^{(-1-i\sqrt{3})t} \\ &= e^{-t}(c_1 e^{i\sqrt{3}t} + c_2 e^{-i\sqrt{3}t}) \\ &= e^{-t}(d_1 \cos(\sqrt{3}t) + d_2 \sin(\sqrt{3}t)) \\ x' &= -e^{-t}(d_1 \cos(\sqrt{3}t) + d_2 \sin(\sqrt{3}t)) \\ &\quad + e^{-t}(-\sqrt{3}d_1 \sin(\sqrt{3}t) + \sqrt{3}d_2 \cos(\sqrt{3}t)) \\ &= e^{-t}((-d_1 + \sqrt{3}d_2) \cos(\sqrt{3}t) + (-d_2 - \sqrt{3}d_1) \sin(\sqrt{3}t)) \end{aligned}$$

← lecture 15

Spring-mass systems (cont.)

Example 2 (cont.)

Initial conditions:

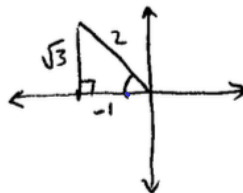
$$\begin{cases} x(0) = 1 \implies 1 = d_1 + 0 \implies d_1 = 1 \\ \underline{x'(0) = -2} \implies \underline{-2 = -d_1 + \sqrt{3}d_2} \implies \underline{d_2 = -1/\sqrt{3}} \end{cases}$$

Hence,

$$\begin{aligned} x(t) &= e^{-t} \left(\cos(\sqrt{3}t) - \frac{1}{\sqrt{3}} \sin(\sqrt{3}t) \right) \leftarrow \\ &= e^{-t} \frac{2}{\sqrt{3}} \left(\frac{\sqrt{3}}{2} \cos(\sqrt{3}t) - \frac{1}{2} \sin(\sqrt{3}t) \right). \end{aligned}$$

Let $\sin \phi = \frac{\sqrt{3}}{2}$ and $\cos \phi = -\frac{1}{2}$ then $\phi = \pi - \frac{\pi}{3} = \frac{2\pi}{3}$ and


$$x(t) = \frac{2}{\sqrt{3}} e^{-t} \sin \left(\sqrt{3} \left(t - \frac{4\pi}{3\sqrt{3}} \right) \right).$$



Task: Write $x(t) = e^{-t}(\cos \sqrt{3}t - \frac{1}{\sqrt{3}} \sin \sqrt{3}t)$ in Amplitude-Phase form.

$$A = \sqrt{c_1^2 + c_2^2} = \sqrt{1 + 1/3} = 2/\sqrt{3}$$

$$\Rightarrow x(t) = 2/\sqrt{3} e^{-t} \left(\underset{\substack{\uparrow \\ \sin \phi}}{\frac{\sqrt{3}}{2} \cos \sqrt{3}t} - \underset{\substack{\uparrow \\ \cos \phi}}{\frac{1}{2} \sin \sqrt{3}t} \right)$$

$$\begin{aligned} \Rightarrow \sin \phi &= \sqrt{3}/2 \\ \cos \phi &= -1/2 \\ \tan \phi &= -\sqrt{3} \end{aligned}$$

$$\Rightarrow \phi = -\pi/3 + \pi = 2\pi/3$$

$$\begin{aligned} \Rightarrow x(t) &= 2/\sqrt{3} e^{-t} \sin(\sqrt{3}t + 2\pi/3) \\ &= 2/\sqrt{3} e^{-t} \sin(\sqrt{3}t + 2\pi/3 - 2\pi) \\ &= 2/\sqrt{3} e^{-t} \sin(\sqrt{3}t - 4\pi/3) \\ &= \underline{2/\sqrt{3}} e^{-t} \sin(\sqrt{3} \left(\underset{\uparrow}{t} - \underline{\frac{4\pi}{3\sqrt{3}}} \right)) \end{aligned}$$

Spring-mass systems (cont.)

Example 2 (cont.)

