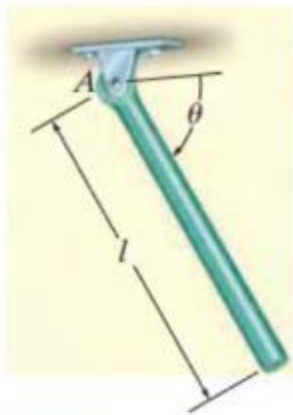


1. The slender rod shown has a mass m and length l and is released from rest when $\theta = 0^\circ$. Determine the horizontal and vertical components of force which the pin at A exerts on the rod at the instant $\theta = 90^\circ$. (20pts)



SOLUTION

Free-Body and Kinetic Diagrams. The free-body diagram for the rod in the general position θ is shown in Fig. 17-17b. For convenience, the force components at A are shown acting in the n and t directions. Note that α acts clockwise and so $(\mathbf{a}_G)_t$ acts in the $+t$ direction.

The moment of inertia of the rod about point A is $I_A = \frac{1}{3}ml^2$.

Equations of Motion. Moments will be summed about A in order to eliminate A_n and A_t .

$$+\curvearrowright \Sigma F_n = m\omega^2 r_G; \quad A_n - mg \sin \theta = m\omega^2(l/2) \quad (1)$$

$$+\checkmark \Sigma F_t = m\alpha r_G; \quad A_t + mg \cos \theta = m\alpha(l/2) \quad (2)$$

$$\zeta + \Sigma M_A = I_A \alpha; \quad mg \cos \theta(l/2) = \left(\frac{1}{3}ml^2\right)\alpha \quad (3)$$

Kinematics. For a given angle θ there are four unknowns in the above three equations: A_n , A_t , ω , and α . As shown by Eq. 3, α is *not constant*; rather, it depends on the position θ of the rod. The necessary fourth equation is obtained using kinematics, where α and ω can be related to θ by the equation

$$(\zeta +) \quad \omega d\omega = \alpha d\theta \quad (4)$$

Note that the positive clockwise direction for this equation *agrees* with that of Eq. 3. This is important since we are seeking a simultaneous solution.

In order to solve for ω at $\theta = 90^\circ$, eliminate α from Eqs. 3 and 4, which yields

$$\omega d\omega = (1.5g/l) \cos \theta d\theta$$

Since $\omega = 0$ at $\theta = 0^\circ$, we have

$$\int_0^\omega \omega d\omega = (1.5g/l) \int_0^{90^\circ} \cos \theta d\theta$$

$$\omega^2 = 3g/l \quad \longrightarrow \quad \text{可以直接使用能量守恒得到, 不用积分}$$

Substituting this value into Eq. 1 with $\theta = 90^\circ$ and solving Eqs. 1 to 3 yields

$$\alpha = 0$$

$$A_t = 0 \quad A_n = 2.5mg$$

Ans.

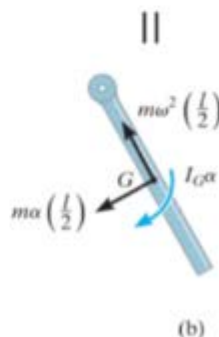
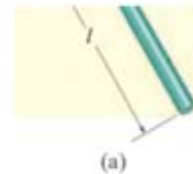
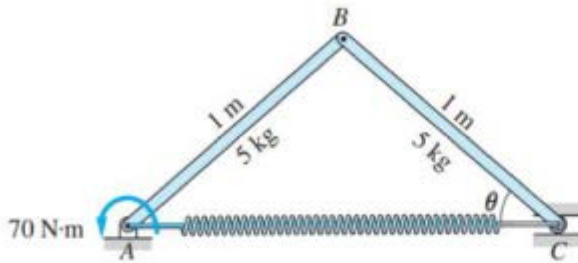


Fig. 17-17

2. The system, which lies in the vertical plane, consists of the 5-kg homogeneous bars AB and BC and the spring AC. The free length of the spring is 1 m, and its stiffness is 200 N/m. A constant 70-N · m couple acts on bar AB. Determine

- (a) the work done on the bars as θ changes from 0 to 45°.
 (b) The velocity of C at the instant. (30pts)



3# Dgn 18-3

$V_{A2} = \omega_{BC} \sqrt{1^2 + 0.5^2} = 1.118 \omega_{BC}$
 $V_{C1} = \omega_{AB} \cdot \frac{1}{2} = \frac{1}{2} \omega_{AB}$
 $V_B = \omega_{BC} \cdot 1 = \omega_{AB} \cdot 1$
 $\Rightarrow \omega_{AB} = \omega_{BC}$

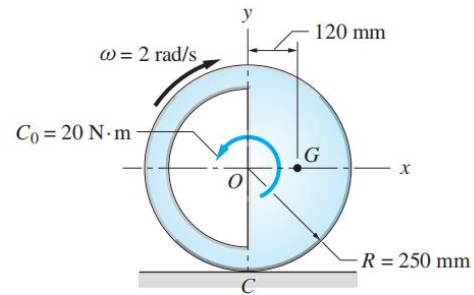
When $\theta = 0^\circ$, $l = 2\text{ m}$, $\Delta l_1 = 2 - 1 = 1\text{ m}$
 when $\theta = 45^\circ$, $l = \sqrt{2}\text{ m}$, $\Delta l_2 = (\sqrt{2} - 1)\text{ m}$.

$U_1 = M(\theta_2 - \theta_1) = 70 \left(\frac{\pi}{4} - 0 \right) = 54.9779\text{ J}$
 $U_2 = \frac{1}{2} k \Delta l_1^2 - \frac{1}{2} k \Delta l_2^2 = \frac{1}{2} \times 200 \times [1^2 - (\sqrt{2} - 1)^2] \approx 82.8427\text{ J}$
 $U_3 = -2mgh = -2 \times 5 \times 9.81 \times \frac{1}{2} \times \frac{\sqrt{2}}{2} \approx -34.0836\text{ J}$
 $U_{\text{total}} = U_1 + U_2 + U_3 = 103.1370\text{ J}$

$T_1 + U_{1 \rightarrow 2} = T_2$
 $0 + 103.1370 = \frac{1}{2} m V_{C1}^2 + \frac{1}{2} I_A \omega_{AB}^2 + \frac{1}{2} m V_{A2}^2 + \frac{1}{2} I_C \omega_{BC}^2$
 $\frac{1}{2} \times 5 \times (0.5 \omega_{BC})^2 + \frac{1}{2} \times \frac{1}{12} \times 5 \times 1^2 \times \omega_{BC}^2 + \frac{1}{2} \times 5 \times (1.118 \omega_{BC})^2 + \frac{1}{2} \times \frac{1}{12} \times 5 \times 1^2 \times \omega_{BC}^2$
 $= 103.1370$
 $\Rightarrow \omega_{BC} \approx 4.9753\text{ rad/s (2)}$
 $v_C = \omega_{BC} \cdot CD$

图上D是CB杆瞬心
 $v_C = \omega_{BC} \cdot CD$

3. The 40-kg unbalanced wheel is rolling without slipping under the action of a counterclockwise couple $C_0 = 20 \text{ N} \cdot \text{m}$. When the wheel is in the position shown, its angular velocity is $\omega = 2 \text{ rad/s}$ clockwise. For this position, calculate the angular acceleration α and the forces exerted on the wheel at C by the rough horizontal plane. The radius of gyration of the wheel about its mass center G is $k_z = 200 \text{ mm}$. (20pts)



FBD

$C_0 = 20 \text{ N} \cdot \text{m}$ $m = 40 \text{ kg}$
 $mg = 40 \times 9.81 = 392.4 \text{ N}$
 N_C, F_C : unknown

KD

$\omega = 2 \text{ rad/s}$ (clockwise)
 $k_z = 0.2 \text{ m}$
 $e = 0.12 \text{ m}$
 $R = 0.25 \text{ m}$
 $I_G = m k_z^2 = 1.6 \text{ kg} \cdot \text{m}^2$
 α : unknown

C : ICR (Instantaneous Center of Rotation) $v_C = 0$
 $a_C \neq 0$ (unknown)

O : $\vec{v}_O = \vec{\omega} \times \vec{r}_{OC}$
 $= -2 \vec{k} \times 0.25 \vec{j} = 0.5 \vec{i} \text{ m/s}$
 (up) (\rightarrow)

$\vec{a}_O = \vec{\alpha} \times \vec{r}_{OC} = 0.25 \vec{i} \text{ m/s}^2$
 (up) (\rightarrow)

$\vec{a}_G = \vec{a}_O + \vec{\alpha} \times \vec{r}_{OG} - \omega^2 \vec{r}_{OG}$
 $= 0.25 \vec{i} + (\alpha \vec{k} \times 0.12 \vec{j}) - 2^2 \cdot 0.12 \vec{i}$
 $= (0.25\alpha - 0.48) \vec{i} - 0.12\alpha \vec{j}$
 $a_{Gx} = 0.25\alpha - 0.48 \text{ (} \rightarrow \text{) m/s}^2$
 $a_{Gy} = 0.12\alpha \text{ (} \downarrow \text{) m/s}^2$

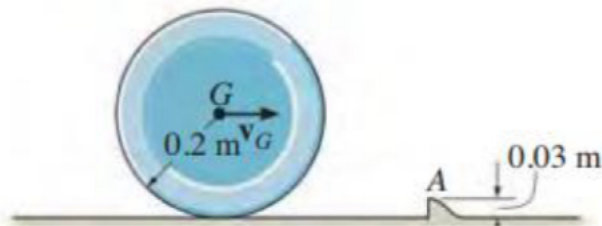
Eqs of Motion

$\rightarrow \sum F_x = m a_{Gx}$ $F_C = 40(0.25\alpha - 0.48) = 10\alpha - 19.2$ ①
 $\downarrow \sum F_y = m a_{Gy}$ $mg - N_C = 40 \times 0.12\alpha$ $N_C = 392.4 - 4.8\alpha$ ②
 $\sum M_G(F) = I_G \alpha$ $N_C \cdot 0.12 - F_C \cdot 0.25 - C_0 = 1.6\alpha$ ③

① ② ③ $\rightarrow \alpha = 6.82 \text{ rad/s}$ $31.988 / 4.676$

$N_C = 359.7 \text{ N (} \uparrow \text{)}$ $F_C = 49.0 \text{ N (} \rightarrow \text{)}$

4. The 10-kg wheel shown has a moment of inertia $I_G = 0.156 \text{ kg} \cdot \text{m}^2$. Assuming that the wheel does not slip or rebound, determine the minimum velocity v_G it must have to just roll over the obstruction at A. (30pts)



Modeling and Analysis:

- Draw the FBD and Impulse and Momentum

Diagrams for the wheel and apply the conservation of the angular momentum about A.

$$(H_A)_1 = (H_A)_2$$

$$mv_{G1} r' + I_G \frac{v_{G1}}{r} = mv_{G2} r + I_G \frac{v_{G2}}{r}$$

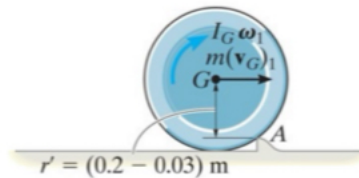
$$mv_{G1} r' + I_G \omega_1 = mv_{G2} r + I_G \omega_2$$

- Kinematics $v_{G1} = \omega_1 r$ $v_{G2} = \omega_2 r$

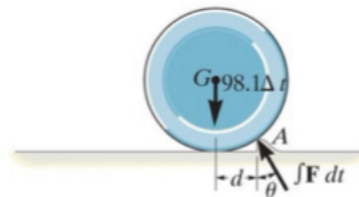
$$v_{G1} \left(mr' + \frac{I_G}{r} \right) = v_{G2} \left(mr + \frac{I_G}{r} \right)$$

- The expression of final angular velocity.

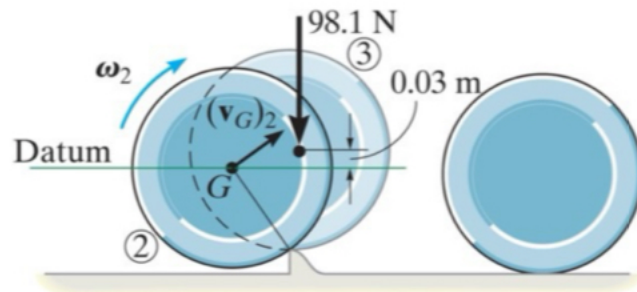
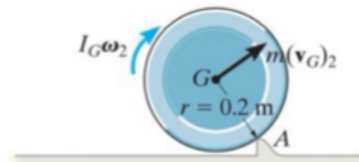
$$\begin{aligned} v_{G2} &= v_{G1} \frac{mr' + I_G / r}{mr + I_G / r} \\ &= v_{G1} \frac{10 \times (0.2 - 0.03) + 0.156 / 0.2}{10 \times 0.2 + 0.156 / 0.2} \\ &= 0.892 v_{G1} \end{aligned}$$



+



|



- Apply the Conservation of Energy, determine the minimum kinetic energy to roll over the obstruction.

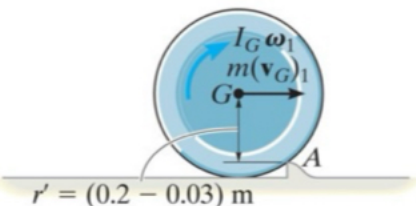
$$T_2 + V_2 = T_3 + V_3$$

$$\frac{1}{2} m v_{G1}^2 + \frac{1}{2} I_G \omega_1^2 + 0 = 0 + m g \times 0.03$$

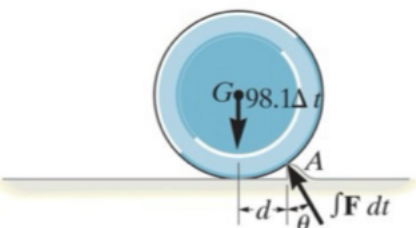
$$v_{G2, \min}^2 = \frac{2 g \times 0.03}{1 + I_G / m r^2} = \frac{2 \times 9.81 \times 0.03}{1 + 0.156 / (10 \times 0.2^2)} = 0.423 (\text{m/s})^2$$

$$v_{G2, \min} = 0.651 \text{ m/s}$$

$$v_{G1, \min} = v_{G2, \min} / 0.892 = 0.730 \text{ m/s} \quad (\rightarrow)$$



+



|

