Politechnico Di Milano

WEEK 9

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Task 1 Use a weather forecast website, and utilize the psychrometric chart and the formula we went through in the class to determine the absoloute humidity, the wet-bulb temperature and the mass of water vapour in the air in ClassRoom A (Aula A) of Piacenza campus in the moment that you are solving this exercise (provide the inputs that you utilized)

Umidità: Relative humidity, Pressione atmosferica: Air total pressure (1 hPa: 0.1 kPa), Temperatura effettiva: temperature to be utilized.

Monday 16 December

		Il tempo oggi in Piacenza Lunedi, 16 Dicembre 2019						
	22:00	23:00						
	∆ ∆ LightRain	∆ ∆ LightRain						
Temperatura effettiva	6°C	7°C						
Temperatura percepita	6°C	7°C						
Precipitazioni	0 mm	0 mm						
Umidità	96 %	95 %						
Pressione atmosferica	1021 hPa	1020 hPa						
Intensità del vento	3 km/h	2 km/h						
Direzione del vento	\hookrightarrow	\hookrightarrow						
	0	0						

Relative humidity: 95 %

Total air pressure: 102.1 kpa

Effective Temperature: 6 C

Absolute humidity: 0.005 Kg _{vapor} / Kg _{dry air}

Wet bulb temperature: 5

$$M_V = P_V V / R_V T$$

$$P_{Total} = P_a + P_v$$

$$\omega = 0.622 \text{ Pv} / \text{P-Pv}$$

Classroom A= 6*10*6= 360 m²

$$R_{v}$$
= 0.4615

$$M_V = (0.814*360)/(0.4615*279)$$

Task 2 Utilize the same methodology we went through in the class and determine the sensible and latent load corresponding to internal gains, the ventilation, and the infiltration in a house with a *good* construction quality and with the same geometry as that of the example which is located in Brindisi, Italy

				BRINDISI, Italy											WMO#:	163200	
	Lat	40.65N	Long:	17.95E	Elev:	10	StdP:	101.2		Time Zone:	1.00 (EU	W)	Period	86-10	WBAN:	99999	
Annual Heating and Humidification Design Conditions																	
					Humidification DP/MCDB and HR							Coldest month WS/MCDB MCWS					
	Coldest Heating DB			99.6%			99% 0.4%					to 99.6% DB					
	Month	99.6%	99%	DP	HR	MCDB	DP	HR	MCDB	WS	MCDB	WS	MCDB	MCWS	PCWD		
	(a)	(b)	(c)	(d)	(0)	(f)	(g)	(h)	(i)	(1)	(k)	(1)	(m)	(n)	(0)		
(1)	2	2.9	4.1	-5.1	2.5	7.2	-3.0	3.0	7.4	13.4	10.2	12.4	10.6	3.4	250		(1)
	Annual Co	oling, Deh	umidificatio	on, and Enth	alpy Desigr	Condition:	•										
	Hottest	Hottest Month	-	4%		0B/MCWB %	24	2% 0.4%			Evaporation WB/MCDB 1% 2%				MCWS/PCWD to 0.4% DB		
	Month	DB Range	DB	MCWB	DB	MCWB	DB 27	MCWB	WB	MCDB	WB	MCDB	WB	MCDB	MCWS	PCWD	
	(a)	(b)	(c)	(d)	(e)	(f)	(g)	(h)	(i)	(j)	(k)	(1)	(m)	(n)	(0)	(p)	
(2)	8	7.1	32.8	23.6	31.1	24.3	29.9	24.3	27.2	29.7	26.3	29.0	25.6	28.3	4.2	180	(2)
				Dehumidification DP/MCDB and HR Enthalpy/MCDB												Hours	
		0.4%			1%	000 0110 111		2%		0.4	4%		%	2	1%	8 to 4 &	
	DP	HR	MCDB	DP	HR	MCDB	DP	HR	MCDB	Enth	MCDB	Enth	MCDB	Enth	MCDB	12.8/20.6	
	(a)	(b)	(c)	(d)	(0)	(f)	(g)	(h)	(i)	(j)	(k)	(1)	(m)	(n)	(0)	(P)	
(3)	26.3	21.8	29.2	25.4	20.7	28.5	24.7	19.7	27.9	86.0	30.1	82.2	29.1	78.5	28.3	1236	(3)
Extreme Annual Design Conditions																	
	Extr	eme Annual	WS	Extreme Annual DB				n-Year Return Period Values of Extreme I									
1% 2.5% 5%			Max WB	Min	an Max	Standard Min	deviation Max	n=5 Min	years Max	n=10 Min	years Max	n=20 Min	years Max	n=50 Min	years Max		
	(a)	(b)	(c)	(d)	(e)	(f)	(g)	(h)	(i)	(j)	(k)	(1)	(m)	(n)	(o)	(p)	
(4)	11.3	9.9	8.7	31.4	0.4	37.3	1.4	3.0	-0.6	39.4	-1.4	41.1	-2.2	42.8	-3.2	44.9	(4)

$$Q_{ig, s}$$
=136+2.2 A_{cf} +22 Noc =136+2.2*200+22*2=620 W

$$AL=A_{es}*A_{ul}=344*1.4=481.6 \text{ cm}^2$$

 $T_{cooling}$ =24 °C, and heating temperature $T_{heating}$ =20 °C

$$\Delta$$
 $T_{cooling} \!\!=\!\! 31.1~^{\circ}\text{C}$ –24 $^{\circ}\text{C} \!\!=\!\! 7.1~^{\circ}\text{C} \!\!=\!\! 7.1~\text{K}$

$$\Delta$$
 T_{heating}=20 °C -(-4. 1 °C)=24.1 °C=24.1 K

$$DR = 7.1 \, ^{\circ}C = 7.1 \, K$$

 $Q_{i,\ heating} = A_L * IDF_{heating} = 481.6 * 0.073 \approx 35.157 L/s$

 $Q_{i,\;cooling}\!\!=\!\!A_L*IDF_{cooling}\!\!=\!\!481.6*0.033\!\!\approx\!\!15.893L/s$

 $Q_v = 0.05A_{cf} + 3.5(N_{br} + 1) = 0.05*200 + 3.5*(1+1) = 17L/s$

 $Q_{i-v,heating} = Q_{i,\;heating} + Q_{v} \approx 35.157 + 17 = 52.157 L/s$

 $Q_{i-v, cooling} = Q_{i, cooling} + Q_{v} \approx 15.893 + 17 = 32.893 L/s$

 $C_{sensible}$ =1.23, C_{latent} =3010, $\Delta\omega_{Cooling}$ =0.0039

 $Q_{inf-ventilation\ cooling\ sensible} = C_{sensible}Q_{i-v}, \\ cooling\ \Delta T_{Cooling} \approx 1.23\ *32.893*7.1 \approx 287.25\ W$

 $Qinf-_{ventilation\;cooling\;latent} = C_{latent}Q_{i-v,\;cooling} \quad \Delta\omega_{Cooling} \approx 3010*32.893*0.0039 \approx 386.13\;W$

 $Qinf-_{ventilation\ heating\ sensible} = C_{sensible}Q_{i-v,\ heating} \quad \Delta T_{heating} \approx 1.23\ *52.157*24.1 \approx 1546.09\ W$