# Low-pressure Hydrogen flow through the UK Gas Distribution Network Supplementary Appendices

Michael Sargent (Da,\*, Philip Sargent (Da

<sup>a</sup>Cambridge Energy UK, 27 Greville Road, Cambridge CB1 3QJ, UK

#### Appendix A. Software used in this study

The python code and input data is published on GitHub: https://github.com/PhilipSargent/h2-in-pipes under the MIT open source license[1].

## Appendix B. Compressibility

All calculations in this paper have used an equation of state and mixing rules appropriate to the pressure and temperature for each pure gas or blend studied. Individual corrections are small, but they multiply together to make a significant difference.

To calculate the compression factor as a function of temperature and pressure, and for gases such as natural gas composed of several different compounds, one needs an equation of state. The low pressures and ambient temperatures of the distribution grid mean that several different equations of state are all sufficiently accurate.

Lozana  $et\ al.$  have recently reviewed[2] the relevant equations of state and there are a bewildering variety of suitable functions.

In this paper we use the 1978 Peng-Robinson equation of state[3, 4], with temperature dependent binary interaction parameters and no volume translation corrections. For the exact calculation method used in this paper, see the code[1].

# Appendix C. Upgrading UK Boilers

In the most recent English housing survey[5], approximately one tenth (7/(59+7)) of domestic boilers on pipedgas were non-condensing and nine-tenths condensing: "the proportion of dwellings with a standard boiler decreased from 9% in 2020 to 7% in 2021, while the proportion with a condensing-combination boiler has increased from 57% to 59% in the same period".

\*Corresponding author: Michael Sargent

Email addresses: michael.sargent@cambridgeenergy.uk (Michael Sargent (b)), philip.sargent@cambridgeenergy.uk (Philip Sargent (b))

URL: 0000-0001-9129-2990 (Michael Sargent  $\bigcirc$  ), 0000-0002-0968-4467 (Philip Sargent  $\bigcirc$  )

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However there are no statistics on how many of the condensing boilers have correctly-adjusted  $50^{\circ}\mathrm{C}$  return-flow settings with appropriate weather compensation. Anecdotally, the proportion is very small, very likely less than 5%. So a reasonable estimate would be that the return temperature is set to  $50^{\circ}\mathrm{C}$  for 5% of the condensing boilers and to  $70^{\circ}\mathrm{C}$  for 95% of them. There is no condensing for the non-condensing boilers so their efficiency is simply that at a 'condensation' temperature of  $100^{\circ}\mathrm{C}$ .

We will assume

- That 4% of boilers are already perfectly adjusted, and the only change will be the multiplier of  $0.974\times$  from the change in fuel condensation behaviour
- That 86% of boilers will go from a condensing temperature of 70°C (87.44% with NG) to 50°C (93.69% with hydrogen), a multiplier of  $0.933 \times$
- That 10% of boilers will change from an efficiency of 86.20% (non condensing) to a fully-condensing, properly adjusted reflow temperature of  $50^{\circ}$ C (93.69% with hydrogen), a multiplier of  $0.920 \times$

The population weighted efficiency multiplier is thus  $0.933 \times$ .

#### Appendix C.1. Published boiler efficiencies

Boiler efficiency[6] is measured either in a test rig at steady-state load, or estimated as a seasonal average asinstalled in a house with a typical daily heating cycle and weather pattern. Repeated on- and off-periods increase losses[7]. Boiler manufacturer published efficiency values may be legally-required in some jurisdictions to be a seasonally-adjusted average number.

### Appendix D. Viscosities and temperature

The viscosities of pure components are calculated from experimental data as a least-squares fit to a power law dependence on absolute temperature. Ideal gases have a power law relationship where the exponent is 0.5, the real gases here have exponents between 0.66 and 1.08. The viscosity of the gas mixture is a molecular fraction weighted average [8] of the viscosities of the components.

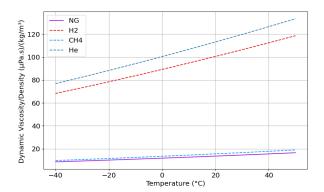


Figure D.1: Plot of kinematic viscosity: the ratio of dynamic viscosity and density, as a function of temperature at 1.06825 bar

Figure D.1 shows the temperature dependence of the kinematic viscosities (the ratio of density to dynamic viscosity) of the natural gas blend, hydrogen and methane. This shows why it is important to do these calculations at a well-chosen reference temperature rather than using textbook values which may have been measured at a variety of different temperatures.

In the UK the ground temperature of the buried LP network is likely to stay in the range 5 to 15°C all year[9].

## Appendix D.1. The Blasius materials property

Note that the term  $\rho^{3/4} \cdot \mu^{1/4}$  in equation (D.1) is a materials' property which depends on temperature and pressure. We define this property as B, the Darcy-Weisbach Reynolds-Blasius parameter in equation (D.2); or abbreviated to the 'Blasius parameter'.

$$\Delta P = \left(\frac{4 \cdot 0.079}{D^{5/4}}\right) \cdot \rho^{3/4} \cdot \mu^{1/4} \cdot v^{7/4} \qquad \text{(D.1)}$$

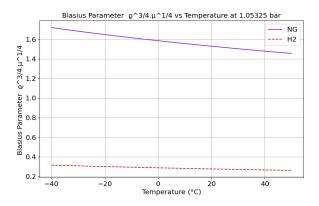


Figure D.2: Blasius Parameter  $\rho^{3/4} \cdot \mu^{1/4} ((kg/m) \cdot (m \cdot s)^{-4})$ 

$$B = \rho^{3/4} \cdot \mu^{1/4} \tag{D.2}$$

where  $\rho$  is the density and  $\mu$  is the dynamic viscosity.

Somewhat surprisingly, this materials' property has only a slight dependence on temperature as the dependencies of density and viscosity act in opposite directions.

Also surprisingly, all the gases in the study have very similar dependence such that the normalised Blasius parameter, where the value for each gas is divided by the value for natural gas, is very nearly independent of temperature, varying less than 1.2% between -40°C and +50°C.

This means that results calculated using the Blasius parameter, the relative pressure drop and the relative compression power requirement, are independent of temperature in the distribution grid.

#### Appendix E. Dew point and partial pressure

Our calculations produce the partial pressure of water vapour in the flue gas, but we need the dew point. The two are related by standard tables of the saturation vapour pressure of water[10]: the pressure at which water vapour is in thermodynamic equilibrium with its condensed state.

The range of atmospheric pressure in the UK means that the dew point of the flue gas varies  $\pm 1.4$ °C, which affects the maximum efficiency.

# Appendix F. The Peng-Robinson calculations

Equations of state for gas mixtures are still an active research area[2] and there is continual development of alpha functions, interaction parameters and mixing rules e.g. Pina-Martinez  $et\ al.$ [11].

Appendix F.1. Calculating the properties of a gas mixture

The higher heat capacity and the mean molecular weight
of the Fordoun natural gas are calculated from a mole fraction weighted sum of those properties of the component
pure gases.

The compressibility and density of the natural gas are calculated for a given temperature and pressure using the Peng-Robinson equation of state:

- 1. The Peng-Robinson[11] parameters a (attraction) and b (repulsion) for each pure component at a given temperature are calculated from the critical temperature  $T_c$ , the critical pressure  $P_c$ , and the *omega* coefficient (a measure of the molecule asphericity) of each pure
- 2. the coefficient b of the mix is the weighted sum of the b values of the component gases, where the weights are the mole fractions.
- 3. The temperature dependent binary interaction parameter k is calculated using the Courtinho method [12] from the b value of each gas in a pair. For an 11 component gas there are 55 pairwise interactions.
- 4. The a values are calculated as the weighted sums of the geometric means of the pair-wise values of a between each pair of components. The weights are the mole factions of each pair multiplied together, times the value (1-k)

- 5. The effective  $T_c$ ,  $P_c$  and omega of the natural gas, the pseudo-component, are calculated from the a and b values for the mix for the given temperature
- 6. the compressibility and thus the density of the natural gas is calculated using the effective  $T_c$ ,  $P_c$  and omega values.

The precise algorithm is in the code on GitHub [1].

#### Appendix G. References

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