

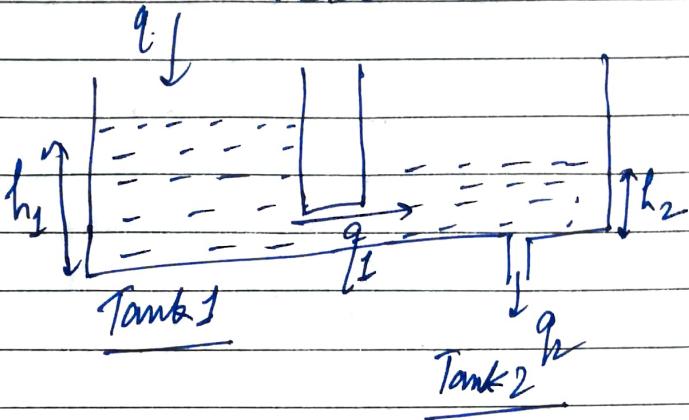
# Assignment 2

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(Q)ii)



$$q_1 = \rho_1 \sqrt{h_1 - h_2}$$

$$q_2 = \rho_2 \sqrt{h_2}$$

$$\frac{\Delta dh_1}{\Delta t} = q - q_1 = q - \rho_1 \sqrt{h_1 - h_2}$$

$$\frac{\Delta dh_2}{\Delta t} = q - q_2 = \rho_1 \sqrt{h_1 - h_2} - \rho_2 \sqrt{h_2}$$

(a)

State Variables: \$h\_1, h\_2,

Input: \$q\$, Output: \$h\_2 = y\$.

Equilibrium Conditions:

\$h\_{20}\$ & \$q\_0\$ are equilibrium states of  
\$h\_2\$ & \$q\$.

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$$\frac{dh_2}{dt} = 0 \Rightarrow q_1 - q_2 = 0$$

$$\Rightarrow P_1 \sqrt{h_1 - h_2} = P_2 \sqrt{h_2}$$

$$\Rightarrow h_{10} - h_{20} = \frac{P_2^2}{P_1^2} h_{20}$$

$$\Rightarrow h_{10} = \left( \frac{P_2^2}{P_1^2} + 1 \right) h_{20}$$

$$\frac{dh_1}{dt} = 0 \Rightarrow q - q_2 = 0$$

$$\Rightarrow q_0 = P_1 \sqrt{h_{20} - h_{10}} = P_2 \sqrt{h_{20}}$$

$$q_0 = P_2 \sqrt{h_{20}}$$

Incremental Transfer function :

(b)

$$f_1 = \frac{dh_1}{dt}, f_2 = \frac{dh_2}{dt}$$

$$\left. \frac{\partial f_1}{\partial h_1} \right|_e = \frac{-P_1}{2\sqrt{h_{10} - h_{20}}} A = \frac{-P_1}{2A P_2 \sqrt{h_{20}}}$$

$$\left. \frac{\partial f_1}{\partial h_1} \right|_e = \frac{-P_1^2}{2A P_2 \sqrt{h_{20}}}$$

$$\left. \frac{\partial f_1}{\partial h_2} \right|_e = \frac{P_1}{2A \sqrt{h_{10} - h_{20}}} = \frac{P_1^2}{2A P_2 \sqrt{h_{20}}}$$

$$\left. \frac{\partial f_1}{\partial h_2} \right|_e = \frac{P_1^2}{2A P_2 \sqrt{h_{20}}}$$

$$\left[ \frac{\partial f}{\partial q} \Big|_c = \frac{1}{A} \right]$$

$$\left[ \frac{\partial f_2}{\partial h_1} \Big|_c = \frac{p_1}{2A\sqrt{h_0-h_2}} = \frac{p_1^2}{2Ap_2\sqrt{h_{20}}} \right]$$

$$\begin{aligned} \frac{\partial f_2}{\partial h_2} \Big|_c &= \frac{-p_1^2}{2Ap_2\sqrt{h_{20}}} - \frac{p_2}{2\sqrt{h_{20}}A} \\ &= \frac{p_2^2 - p_1^2}{2Ap_2\sqrt{h_{20}}} \end{aligned}$$

State Space Rep.

$$\frac{d}{dt} \begin{bmatrix} h_1 \\ h_2 \end{bmatrix} = \begin{bmatrix} \frac{p_1^2}{2Ap_2\sqrt{h_{20}}} & \frac{p_2^2 - p_1^2}{2Ap_2\sqrt{h_{20}}} \\ \frac{p_2}{2\sqrt{h_{20}}} & \frac{p_2^2 - p_1^2}{2Ap_2\sqrt{h_{20}}} \end{bmatrix} \begin{bmatrix} h_1 \\ h_2 \end{bmatrix} + \begin{bmatrix} \frac{1}{4} \\ 0 \end{bmatrix} q$$

$$y = C \begin{bmatrix} 0 & 1 \\ 1 & 0 \end{bmatrix} \begin{bmatrix} h_1 \\ h_2 \end{bmatrix} = C \begin{bmatrix} 0 & 1 \end{bmatrix} \begin{bmatrix} h_1 \\ h_2 \end{bmatrix}$$

$$G(s) = C(SI-A)^{-1} R$$

$$(SI-A)^{-1} = \begin{bmatrix} S + \frac{p_1^2}{2Ap_2\sqrt{h_{20}}} & -\frac{p_1^2}{2Ap_2\sqrt{h_{20}}} \\ -\frac{p_2^2}{2Ap_2\sqrt{h_{20}}} & S + \frac{p_2^2 - p_1^2}{2Ap_2\sqrt{h_{20}}} \end{bmatrix}^{-1}$$

$$= \frac{l}{|\det(A)|} \begin{bmatrix} s + \frac{\rho_1^2 + \rho_2^2}{2Al_2\sqrt{h_{20}}} & \frac{l_1^2}{2Al_2\sqrt{h_{20}}} \\ \frac{\rho_1^2}{2Al_2\sqrt{h_{20}}} & s + \frac{\rho_1^2}{2Al_2\sqrt{h_{20}}} \end{bmatrix}$$

where,  $\det(A) =$

$$\begin{aligned} & \left( s + \frac{\rho_1^2 + \rho_2^2}{2Al_2\sqrt{h_{20}}} \right) \left( s + \frac{\rho_1^2}{2Al_2\sqrt{h_{20}}} \right) - \frac{\rho_1^2}{2Al_2\sqrt{h_{20}}} \cdot \frac{\rho_1^2}{2Al_2\sqrt{h_{20}}} \\ &= s^2 + \frac{\rho_1^2 + 2\rho_2^2}{2Al_2\sqrt{h_{20}}} + \frac{\rho_1^2 \rho_2^2}{4Al_2^2 h_{20}} \\ &= s^2 + \frac{\rho_1^2 + 2\rho_2^2}{2Al_2\sqrt{h_{20}}} + \frac{\rho_1^2}{4A^2 h_{20}} \end{aligned}$$

$$(b/s) = c(SI-A)^{-1}B = \cancel{\text{XXXX}}$$

$$(b/s) = \frac{\frac{l_1^2}{2l_2\sqrt{h_{20}} \times A^2}}{s^2 + \frac{\rho_1^2 + 2\rho_2^2}{2Al_2\sqrt{h_{20}}} s + \frac{l_1^2}{4A^2 h_{20}}}.$$

Transfer function

$$(C) A \frac{dh_1}{dt} = q_0 - \frac{p_1^2}{2p_2 \sqrt{h_{20}}} (h_1, h_2)$$

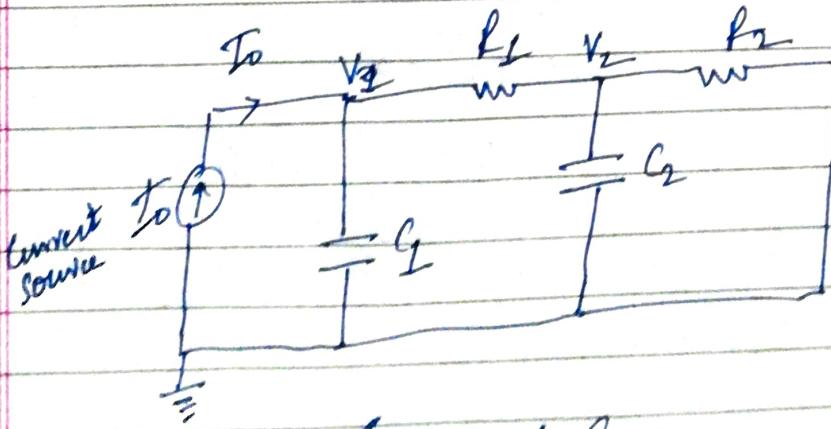
$$A \frac{dh_2}{dt} = \frac{p_1^2}{2p_2 \sqrt{h_{20}}} (h_1 - h_2) - \frac{p_2}{2\sqrt{h_{20}}} \Delta h_{20}$$

$$\text{if } h_1 = V_1 \quad \text{and} \quad q_0 = I_0 \\ h_2 = V_2 \quad A = G = C_2$$

$$\frac{p_1^2}{2p_2 \sqrt{h_{20}}} = \frac{1}{R_1}, \quad \frac{2\sqrt{h_{20}}}{p_2} = \frac{1}{R_2}$$

$$G \frac{dV_1}{dt} = I_0 - R \frac{(V_1 - V_2)}{R_1}$$

$$G \frac{dV_2}{dt} = R \frac{V_1 - V_2}{R_2} - \frac{V_2}{R_2}$$



Electrical Equivalent System.

# Control System Engineering

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(ii)

Given

$$d_0 = 0.5 \text{ cm}$$

$$q_1 = q_2 = \frac{\pi d_0^2}{4} = \frac{\pi \times 0.25 \text{ cm}^2}{4}$$

$$= 0.1963 \text{ cm}^2$$

$$A = 16 \text{ cm}^2$$

$$h_{20} = 10 \text{ cm}$$

$$g = 980 \text{ cm/s}^2$$

$$km = 5 \text{ cm}^3/\text{s}/\nu$$

$$q = 6 \text{ m}^3$$

$$P_1 = q_1 \sqrt{2g} = 0.1963 \sqrt{2 \times 980} = 8.693 \text{ cm}^{2.5}/\nu$$

$$P_2 = q_2 \sqrt{2g} = 0.1963 \sqrt{2 \times 980} \\ = 8.693 \text{ cm}^{2.5}/\nu.$$

$$h_{20} = \frac{P_1^2 + P_2^2}{P_{12}} h_{20} = 2h_{20} = 20 \text{ cm}$$

$$q_0 = P_2 \sqrt{h_{20}} = 8.693 \sqrt{10} = 27.49 \text{ cm}^3/\nu.$$

$$V_o = \frac{q_0}{km} = \frac{27.49}{5} \nu = 5.498 \nu$$

$$\frac{P_1^2}{2A P_2 \sqrt{h_{20}}} = \frac{8.693}{2 \times 16 \sqrt{50}} = 0.086$$

$$\frac{I}{A} = \frac{1}{0.0625}$$

$$\frac{P_1^2 + I^2}{2A P_2 \sqrt{h_{20}}} = \frac{4 \times 8.693}{2 \times 16 \sqrt{50}} = 0.178$$

$$\frac{0.086}{A} = 5.375 \times 10^{-3}$$

$$A = \begin{bmatrix} -0.086 & 0.086 \\ 0.086 & -0.178 \end{bmatrix} \quad B = \begin{bmatrix} 0.0625 \\ 0 \end{bmatrix}$$

$$C = \begin{bmatrix} 0 & 1 \end{bmatrix}$$

$$G(s) = \frac{5.375 \times 10^{-3}}{s^2 + 0.258s + 0.0074}$$

$\omega_n$  —————  $\alpha$  —————  $\zeta$

Rough

$$\omega_n^2 = 0.0074 \quad \zeta = \frac{\pi - \sqrt{8.62^2 - (1-\zeta)^2}}{\sqrt{w_n^2 - \frac{(1-\zeta)^2}{4}}}$$

$$2\zeta \omega_n = 0.258$$

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### Non Linear Model:

$$\frac{Adh_1}{dt} = q = \rho_1 \sqrt{h_1 - h_2}$$

$$16 \frac{dh_1}{dt} = 27.49 - 8.693 \sqrt{h_1 - h_2}$$

$$\boxed{\frac{dh_1}{dt} = 1.719 - 0.5433 \sqrt{h_1 - h_2}}$$

$$\frac{Adh_2}{dt} = \rho_2 \sqrt{h_1 - h_2} - \rho_1 \sqrt{h_2}$$

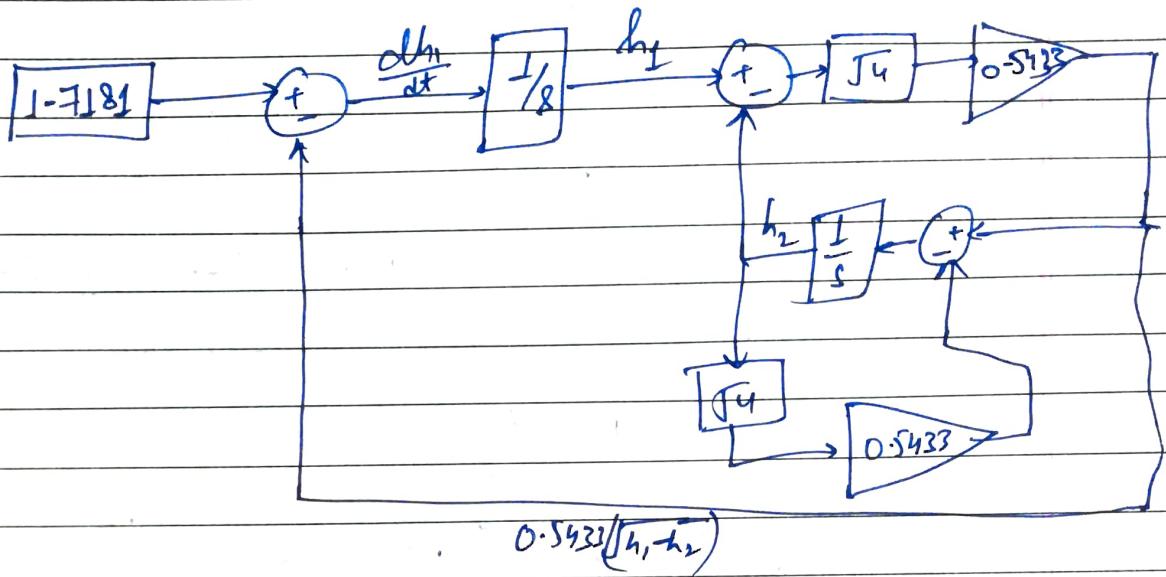
$$16 \frac{dh_2}{dt} = 8.693 \sqrt{h_1 - h_2} - 8.693 \sqrt{h_2}$$

$$\boxed{\frac{dh_2}{dt} = 0.5433 (\sqrt{h_1 - h_2} - \sqrt{h_2})}$$

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## Non Linear Model



## Linear Model:

$$I_0 = \Delta q$$

$$q = C_2 = A = 16$$

$$V_1 = \Delta h_1$$

$$V_2 = \Delta h_2$$

$$R_1 = R_2 = \frac{2q_0}{I^2} = \frac{2 \times 27.49}{8.693^2}$$

$$= 0.7276$$

$$9 \frac{dV_1}{dt} = I_0 - \frac{V_1 - V_2}{R}$$

$$16 \frac{dV_1}{dt} = I_0 - \frac{V_1 - V_2}{0.7276}$$

$$\Rightarrow \boxed{\frac{dV_1}{dt} = 0.0625 I_0 - 0.0859(V_1 - V_2)}$$

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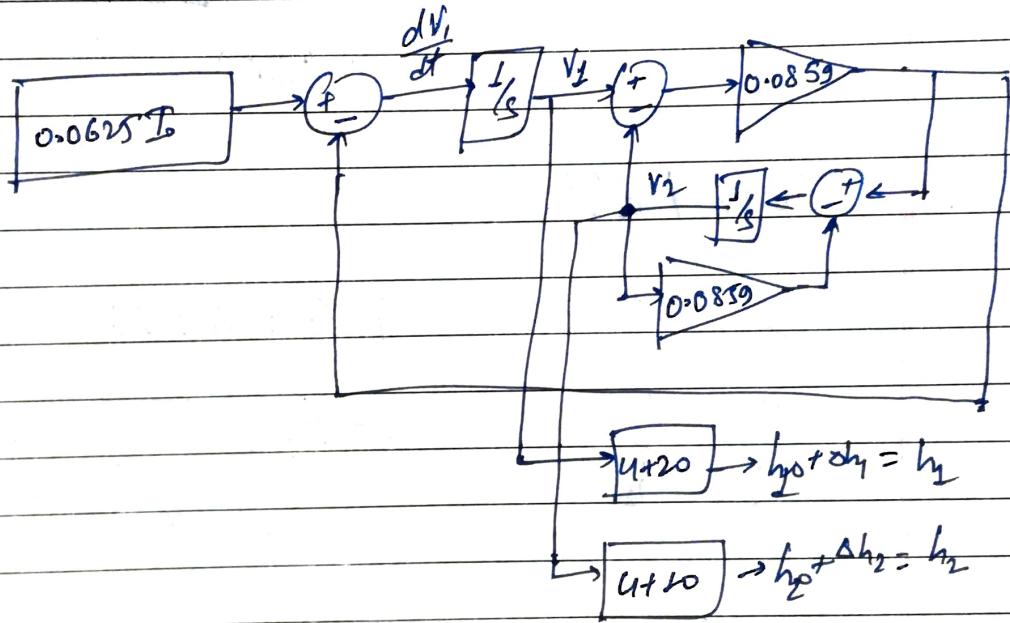
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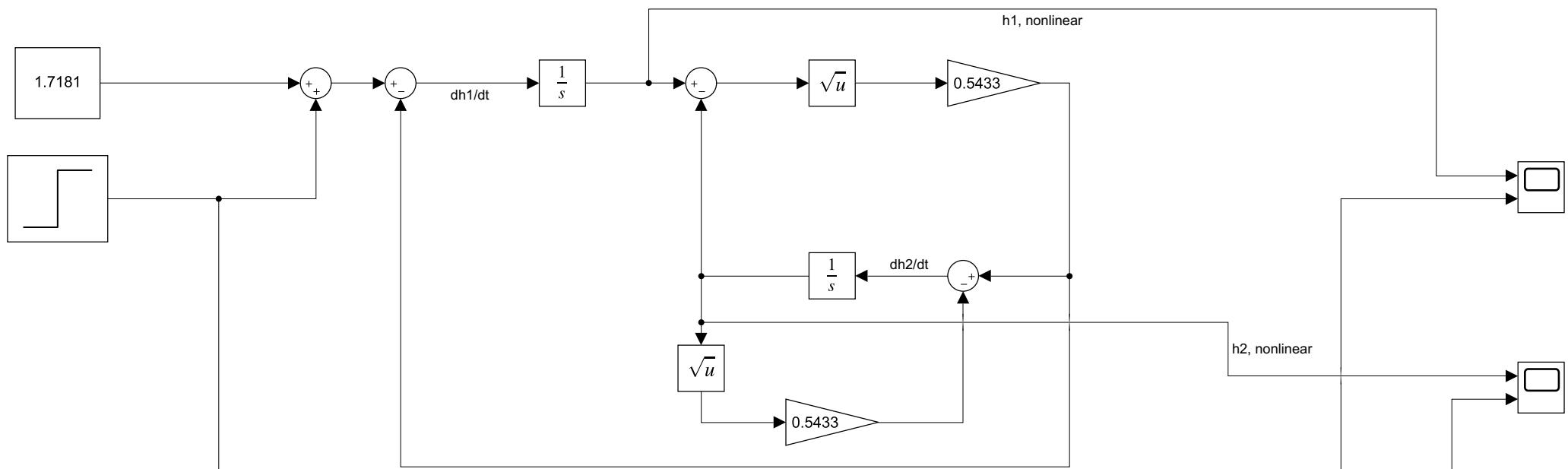
$$C_2 \frac{dV_2}{dt} = \frac{V_1 - V_2}{R_1} - \frac{V_2}{R_2}$$

$$16 \frac{dV_2}{dt} = \frac{V_1 - V_2}{R_1} - \frac{V_2}{0.7276}$$

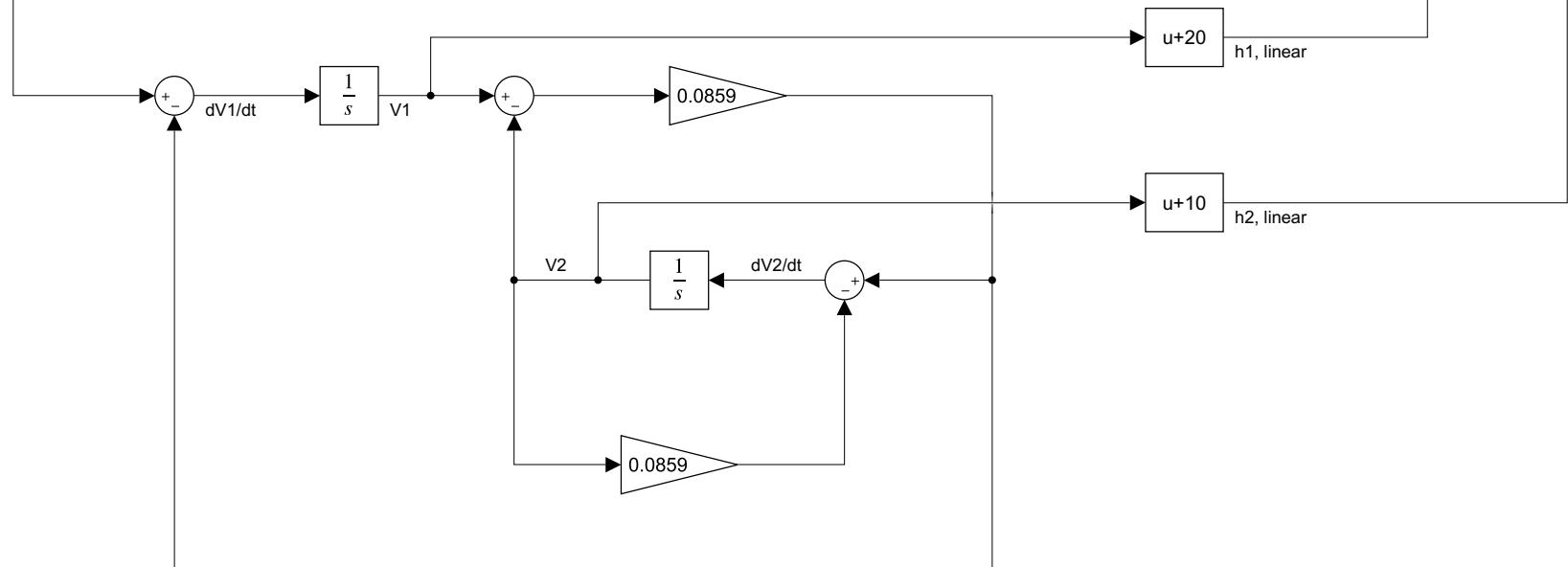
$$\Rightarrow \boxed{\frac{dV_2}{dt} = 0.0859(V_1 - V_2) - 0.0859V_2}$$

Linear Model



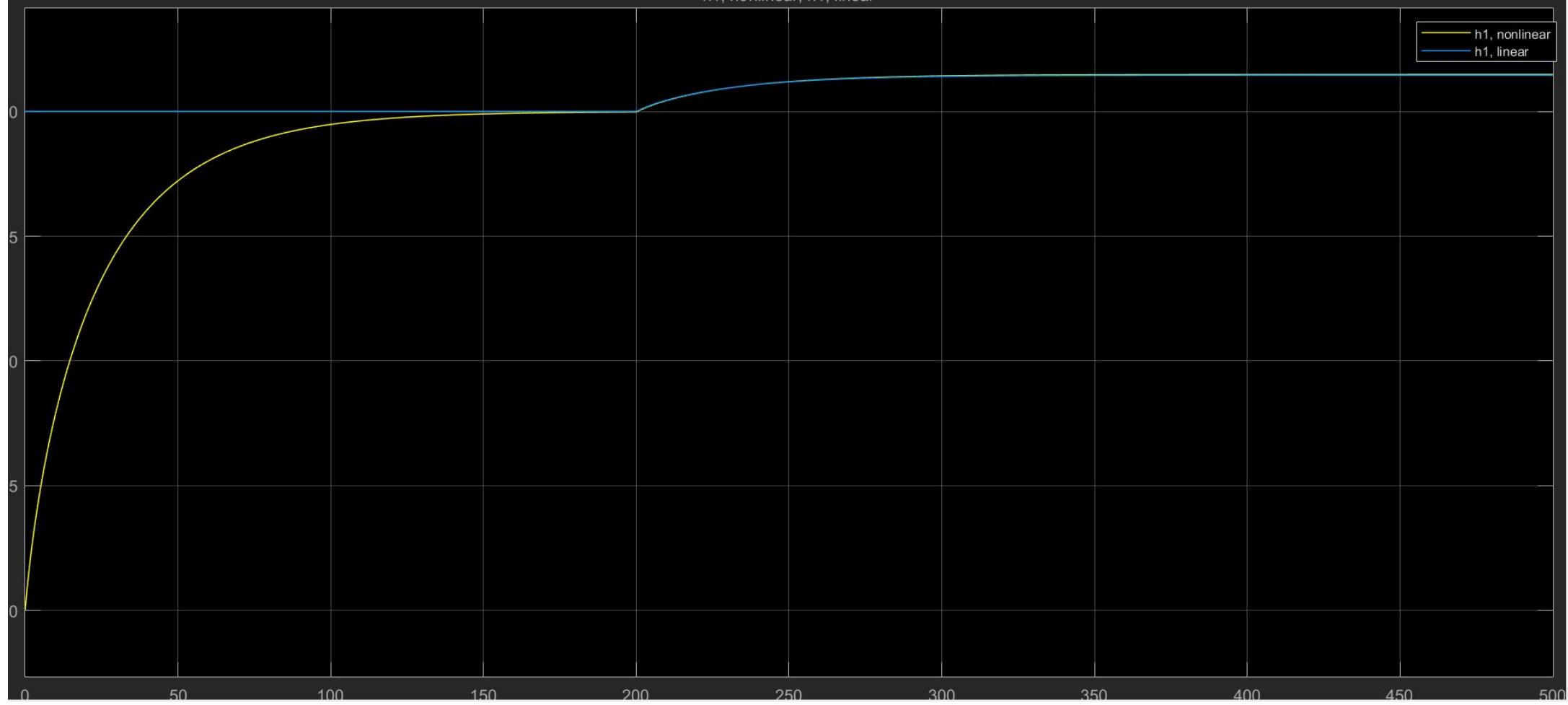


**Non-Linear Model**

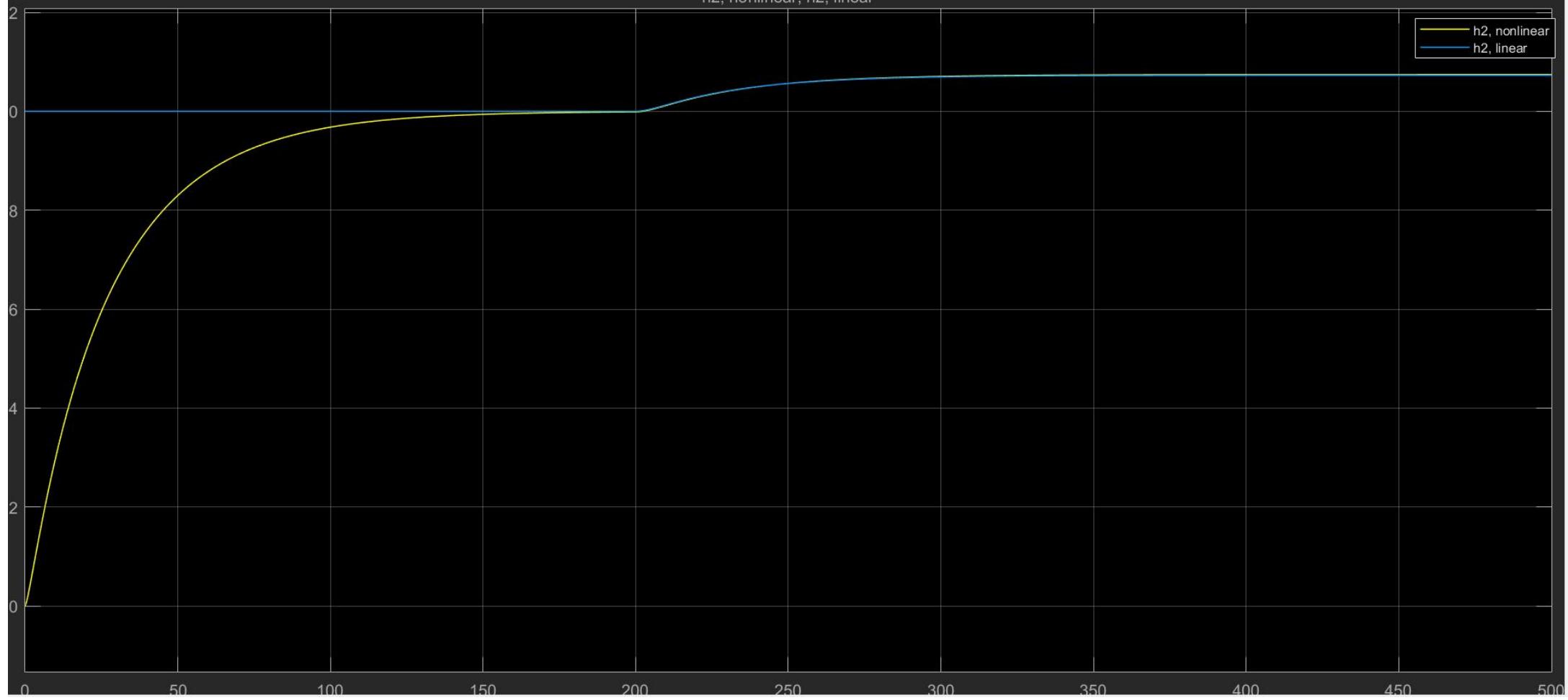


**Linear Model**

# h1, nonlinear, h1, linear

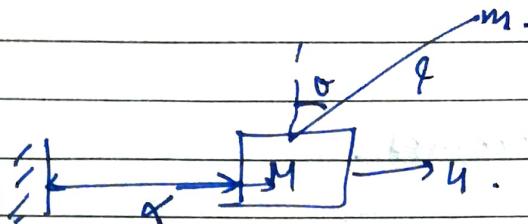


## h2, nonlinear, h2, linear

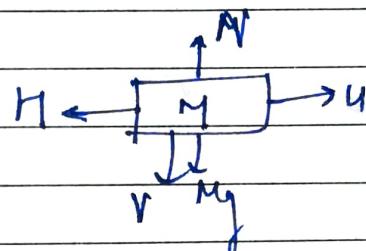


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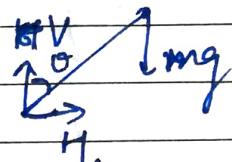
Q2:



FBD of M.



FBD of m+rod.



Force Equation:

$$F - H = M \frac{d^2x}{dt^2} \quad \text{--- (1)}$$

$$F - mg = m \frac{d^2}{dt^2} (l \cos\theta) \quad \text{--- (2)}$$

$$\# H = m \frac{d^2}{dt^2} (l \sin\theta + r) \quad \text{--- (3)}$$

Torque Equation,

Taking I as moment of inertia of bob about its centre of mass, (also considering radius of bob ~~as l~~)

$$(V \sin\theta - H \cos\theta) l = I \frac{d^2\theta}{dt^2} \quad \text{--- (4)}$$

(a) When  $\theta$  being close to 0.

$\sin \theta \approx \theta$ ,  $\cos \theta \approx 1$ .

Then,

Equation (2) becomes.

$$V - mg = 0 \Rightarrow V = mg$$

Eq (3) becomes.

$$H = ml\dot{\theta} + m\ddot{\theta}$$

Eq (4) becomes

$$Vl\dot{\theta} - Hl = I\ddot{\theta}$$

Substituting values from modified eq (2) & (3)  
into Eq (4),

$$mgl\dot{\theta} - ml^2\ddot{\theta} - ml\ddot{\theta} = I\ddot{\theta}$$

$$\Rightarrow mgl\dot{\theta} - ml\ddot{\theta} = (I + ml^2)\ddot{\theta}$$

Neglecting  $I$  in comparison to  $ml^2$ .

$$mgl\dot{\theta} - ml\ddot{\theta} = ml^2\ddot{\theta} \quad \text{--- (5)}$$

$$mgo - ml\ddot{\theta} = ml^2\ddot{\theta} \quad \text{--- (5)}$$

$$u - ml\ddot{\theta} - mi\ddot{i} = mi\ddot{i}$$

$$\boxed{u = ml\ddot{\theta} + (m+M)\ddot{i}} \quad \text{--- (6)}$$

Assuming relaxed initial condition, apply Laplace transformation,

$$U(s) = m\ell s^2 \phi(s) + (M+m)s^2 x(s)$$

$$\phi(s) = \frac{U(s) - (M+m)s^2 x(s)}{m\ell s^2}$$

On Eqn (5)

$$mg\phi(s) - ms^2x(s) = m\ell s^2\phi(s)$$

$$\boxed{\phi(s) = \frac{ms^2x(s)}{mg - m\ell s^2}}$$

$$\frac{ms^2x(s)}{mg - m\ell s^2} = \frac{U(s) - (M+m)s^2 x(s)}{m\ell s^2}$$

$$\Rightarrow U(s) = \frac{m\ell s^2 x(s) + ms^2 x(s)}{mg - m\ell s^2} + (M+m)s^2 x(s)$$

$$\therefore \boxed{\frac{x(s)}{U(s)} = \frac{15^2 - g}{s^2 [M\ell s^2 - (M+m)g]}} = \frac{0.26 s^2}{s^2 (0.6240 s^2 - 26.0680)}$$

$$\frac{\theta(s)}{V(s)} = \frac{m/s^2}{mg - mgs^2} \times \frac{ls^2 - g}{s^2 [Mls^2 + (M+m)g]}$$

$$\left[ \frac{\theta(s)}{V(s)} = \frac{1}{(M+m)g - Mls^2} \right] = \frac{1}{\cancel{26.068} - 0.6240s^2}$$

(b).  $(M+m)\ddot{x} + ml\ddot{\theta} = u \quad \textcircled{7}$

$$\ddot{x} + l\ddot{\theta} = go \quad \textcircled{8}$$

Subtract

Multiply  $m$  eq(8) & subtract from eq(7).

$$M\ddot{\theta} = u - mgo$$

$$\boxed{\ddot{x} = \frac{u - mgo}{M}}$$

$$(7) - (M+m) \textcircled{8}$$

$$-Ml\ddot{\theta} = u - (M+m)go$$

$$\boxed{\ddot{\theta} = \frac{(M+m)go - u}{Ml}}$$

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State Variables :  $x, \dot{x}, \ddot{x}, \ddot{\dot{x}}$       Output :  $\begin{bmatrix} u \\ \dot{x} \end{bmatrix}$

$$\begin{bmatrix} \ddot{x} \\ \ddot{\dot{x}} \\ \ddot{\ddot{x}} \\ \ddot{\dot{\ddot{x}}} \end{bmatrix} = \begin{bmatrix} 0 & 1 & 0 & 0 \\ 0 & 0 & -\frac{mg}{M} & 0 \\ 0 & 0 & 0 & 1 \\ 0 & 0 & 0 & \frac{(M+m)g}{M} \end{bmatrix} \begin{bmatrix} u \\ \dot{x} \end{bmatrix} + \begin{bmatrix} 0 \\ \frac{1}{M} \\ 0 \\ -\frac{1}{M} \end{bmatrix} \quad 4$$

$$y = \begin{bmatrix} 1 & 0 & 0 & 0 \\ 0 & 0 & 1 & 0 \end{bmatrix} \begin{bmatrix} u \\ \dot{x} \\ \ddot{x} \\ \ddot{\dot{x}} \end{bmatrix} + \begin{bmatrix} 0 \\ 0 \end{bmatrix} \quad 4.$$

$$m = 0.26 \text{ kg}$$

$$M = 2.4 \text{ kg}$$

$$l = 0.36 \text{ m}$$

from Matlab,

$$\text{Rank of } [B \ AB \ A^2B \ A^3B] = 4.$$

$$\text{Rank of } \begin{bmatrix} C \\ CA \\ CA^2 \\ CA^3 \end{bmatrix} = 4.$$

$\Rightarrow$  Observability = 4

Controllability = 4.

```

m = 0.26;
M = 2.4;
l = 0.26;
g = 9.8;
A = [0 1 0 0
      0 0 -(m*g)/M 0
      0 0 0 1
      0 0 ((M+m)*g)/(M*l) 0];
B = [0
      1/M
      0
      -1/(M*l)];
C1 = [1 0 0 0];
C2 = [0 0 1 0];
D = [0
      0];

```

```

cont_mat = [B A*B A*A*B A*A*A*B];
controllability = rank(cont_mat)

```

controllability = 4

```

observability_mat_x = [C1; C1*A; C1*A*A; C1*A*A*A;];
obervaility_x = rank(observability_mat_x)

```

obervaility\_x = 4

```

observability_mat_theta = [C2; C2*A; C2*A*A; C2*A*A*A;];
obervaility_theta = rank(observability_mat_theta)

```

obervaility\_theta = 2

∴ Pendulum can be balanced using n.t & feedbacks only no feedbacks.

Because for

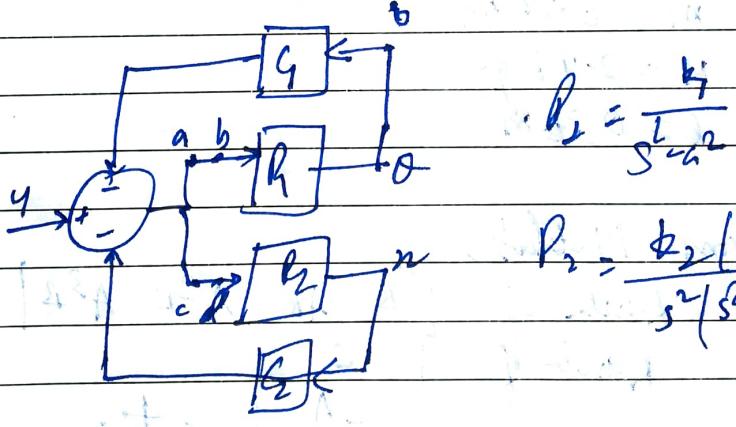
$$G = \begin{bmatrix} 1 & 0 & 0 & 0 \end{bmatrix} \Rightarrow \text{observability} = \text{controllability} = 4$$

$$\& \text{ same when } C_2 = \begin{bmatrix} 0 & 0 & 1 & 0 \end{bmatrix} \Rightarrow$$

$$\begin{aligned} \text{observability} &= 2 \\ \text{controllability} &= 4. \end{aligned}$$

There is a pole zero cancellation in  $G(s)$  which reduced a rank of 2.

(C)



$$P_1 = \frac{b}{s^2 - a^2}$$

$$P_2 = \frac{\phi_2 / (s^2 - b^2)}{s^2 / (s^2 - a^2)}$$

If we break the loop between  $a$  &  $b$ , i.e.

$$\text{loop gain} = \frac{P_1 G}{1 + P_2 (2, 0)}$$

& between  $c$  &  $d$ : c. ?!

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$$\text{Loop gain}_2 = \frac{P_2 G_2}{1 + P_1 G_1}$$

loop gain<sub>1</sub>

$$= \frac{\frac{b_1}{s^2 - a^2} G_1}{1 + G_1 \frac{b_2(s^2 - b^2)}{s^2 - a^2}}$$

loop gain<sub>2</sub>

$$= \frac{b_2(s^2 - b^2) G_2}{s^2(s^2 - a^2) \cdot 1 + G_2 \frac{b_1}{s^2 - a^2}}$$

$$= \frac{b_2 G_2}{(s^2 - a^2) + b_2 G_2 (s^2 - b^2)}$$

$$= \frac{b_2(s^2 - b^2) G_2}{s^2 [s^2 - a^2 + b_2 G_2]}$$

It is

Here loop gain<sub>2</sub> has a right hand plane zero  $s = +b$ .

Therefore we have a robustness issue in case of  $\alpha$ -feedback.

But in case of  $\beta$ -feedback, there is no right hand plane zero, so there is no robustness issue.