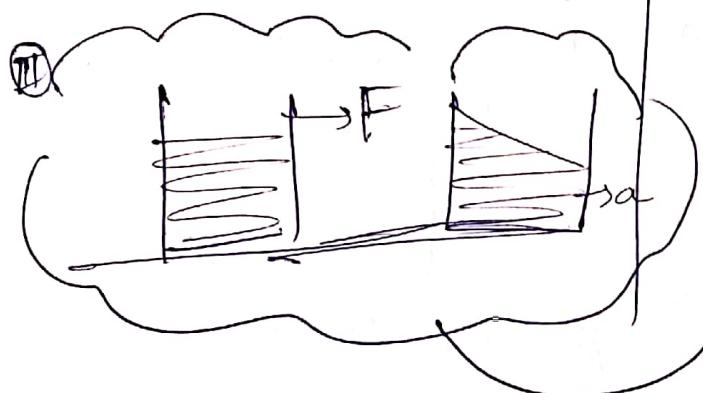
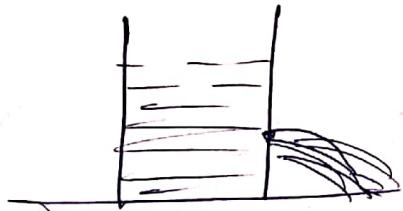


fluid Mechanics

fluid dynamics (Fluid underflow)

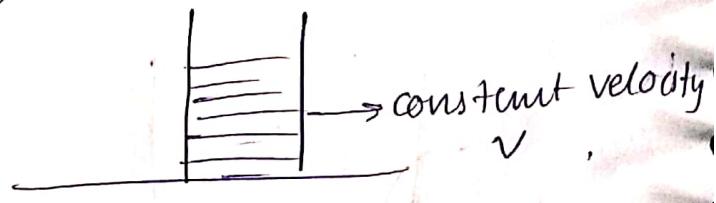


fluid statics

(fluids at rest w.r.t to container)



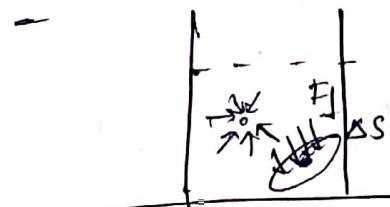
II



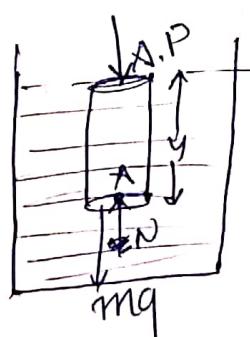
fluid Statics

pressure at a point

- direction of pressure at a point is not defined



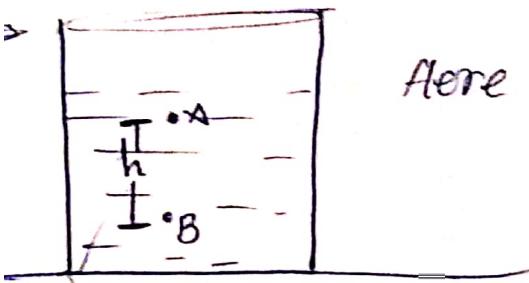
when small area is assumed at a point then direction of pressure is defined.



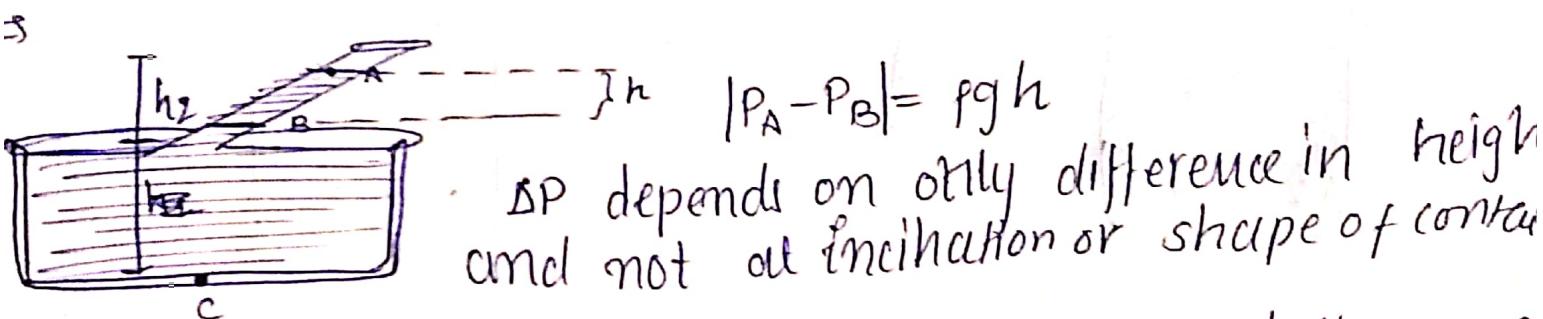
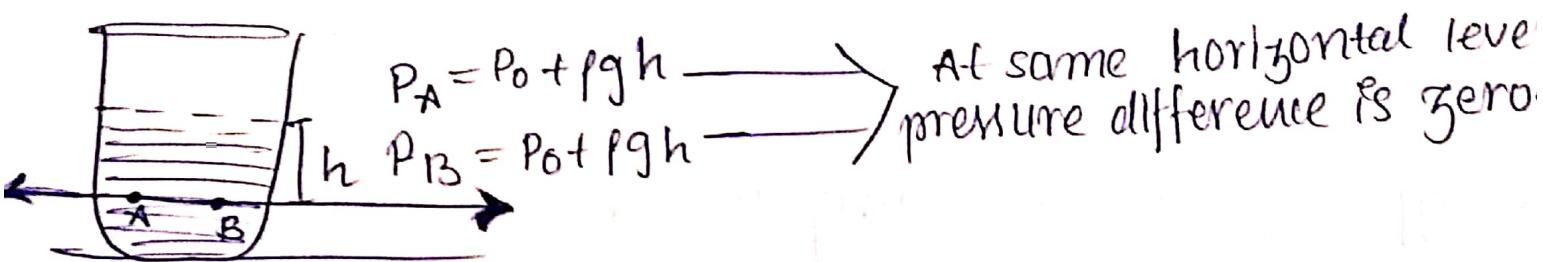
$$P_A = P_0 \cdot \cancel{A} + \cancel{P_0} \cdot y \cdot g$$

$$P_A = P_0 + \cancel{P_0} \cdot y \cdot g$$

absolute pressure

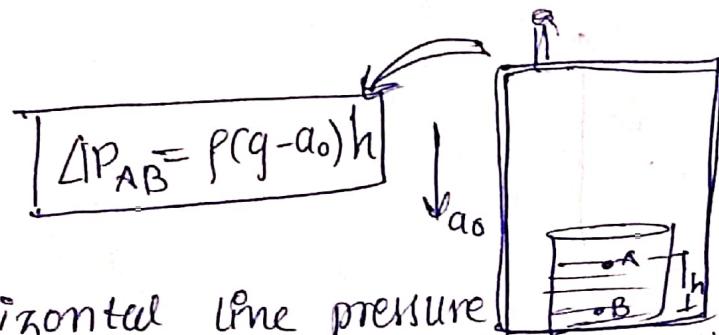
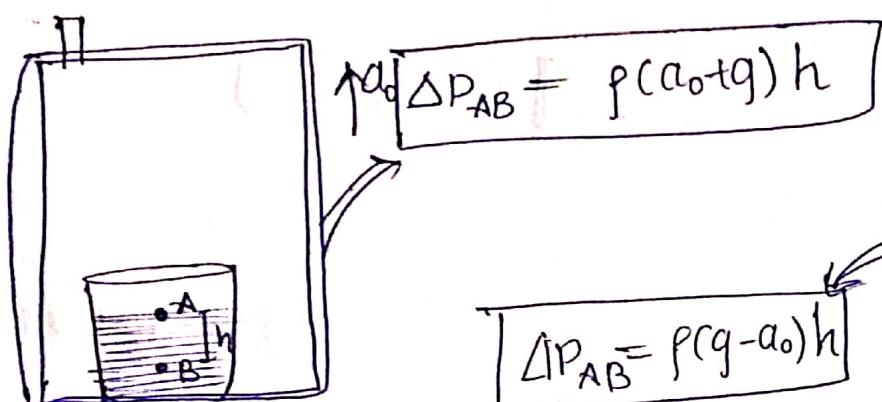


Here $P_B > P_A$
and $P_B - P_A = \rho gh$



$P_C = \rho g (h_1 + h_2) + P_0$ } Absolute pressure at bottommost point.

Finding pressure difference when vessel is accelerating



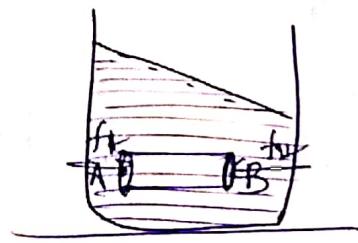
Note :- Along horizontal line pressure remains same if vessel is kept in lift accelerating upward.

If the lift is freely falling under gravity,

$$a_0 = g,$$

$\Rightarrow \Delta P_{AB} = 0 \Rightarrow$ Pressure at all points is same and equal to P_0 if vessel is in atmosphere. Otherwise in vacuum it's zero.

→ If vessel is accelerating horizontally



$$f_r - f_f = m a_0$$

$$P_A - P_B = \rho l a_0$$



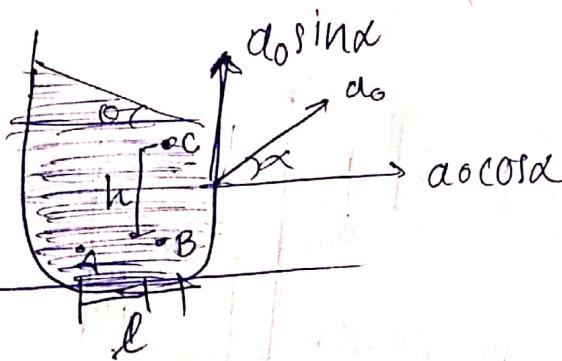
→ Along horizontal pressure will vary here.

All isobaric lines will make an angle $\tan^{-1}(a_0/g)$ with horizontal.

$$\Delta P_{AB} = \rho g (h_1 - h_2) = \rho l a_0$$

$$\frac{a_0}{g} = \frac{l}{h_1 - h_2}$$

$$\Rightarrow a_0 = g \tan \theta$$



$$P_A - P_B = \rho a_0 \cos \theta L$$

$$P_B - P_C = \rho (g + a_0 \sin \theta) h$$

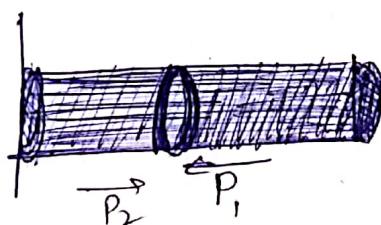
$$P_A - P_C = \rho l (g + a_0 \cos \theta + a_0 \sin \theta) h$$

$$P_A - P_C = \rho a_0 \cos \theta L + \rho (g + a_0 \sin \theta) h$$

$$\tan \theta = \frac{a_0 \cos \theta}{g + a_0 \sin \theta}$$

$$\theta = \tan^{-1} \left(\frac{a_0 \cos \theta}{g + a_0 \sin \theta} \right)$$

→ finding pressure difference in rotating vessel containing fluid.



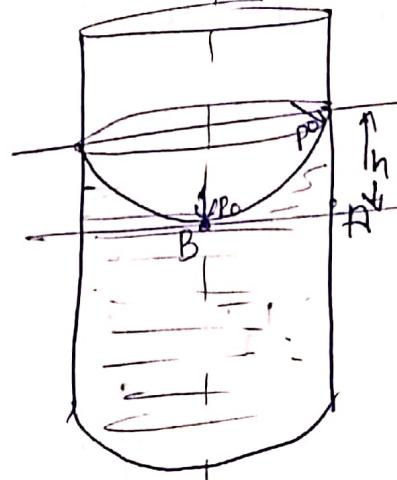
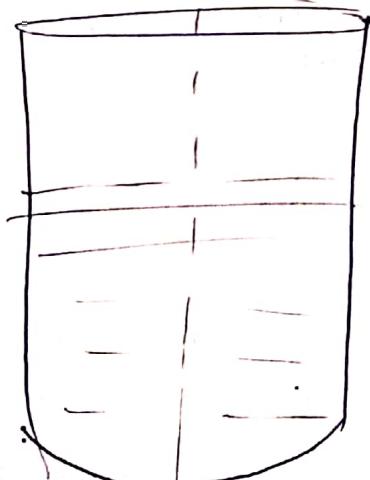
$$\int_{\text{net}} = (P + dp) \Delta s - P \Delta s \\ = dp \cdot \Delta s$$

pressure increases
as we go away from axis.

$$\Rightarrow dm \cdot r \omega = dp \cdot ds \\ \rho \cdot ds \cdot dr \cdot r \omega^2 = dp \cdot ds$$

$$\Rightarrow \int_{P_0}^P = \rho \frac{r^2 \omega^2}{2}$$

$$\Rightarrow P = P_0 + \rho \frac{r^2 \omega^2}{2}$$

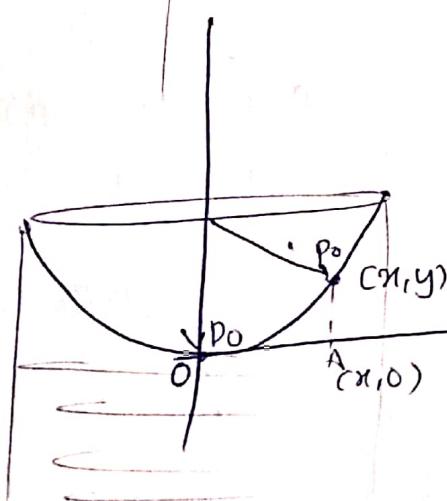


$$P_A - P_0 = \rho g h$$

$$P_A - P_0 = \rho R^2 \omega^2$$

$$\Rightarrow \rho g h = \rho R^2 \omega^2$$

$$\Rightarrow h = \frac{R^2 \omega^2}{2g}$$



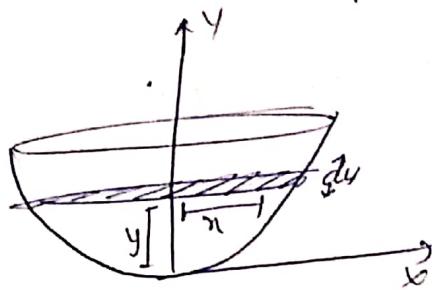
$$P_A - P_0 = \rho g y$$

$$P_A - P_0 = \rho \omega^2 x^2$$

$$\Rightarrow y = \frac{\omega^2 x^2}{2g} = kx^2 \quad \text{where } k = \frac{\omega^2}{2g}$$

⇒ The free surface has parabolic shape.

Volume of parabolic bowl in previous case
(paraboloid)



$$dV = \pi x^2 \cdot dy$$

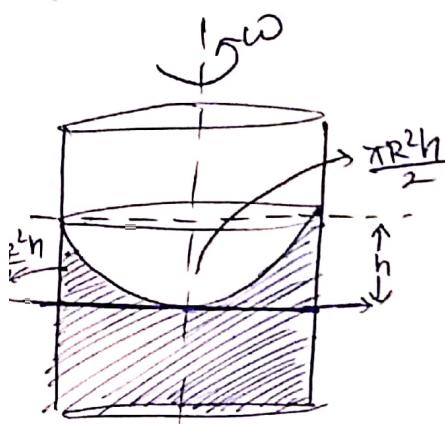
$$\int dV = \int_0^h \pi \left(\frac{2gy}{w^2}\right) \cdot dy$$

$$V = \frac{2g\pi}{w^2} \cdot \left[\frac{h^2}{2}\right]$$

$$= \frac{g\pi}{w^2} \cdot \left[\frac{r^2 w^2}{2g}\right]$$

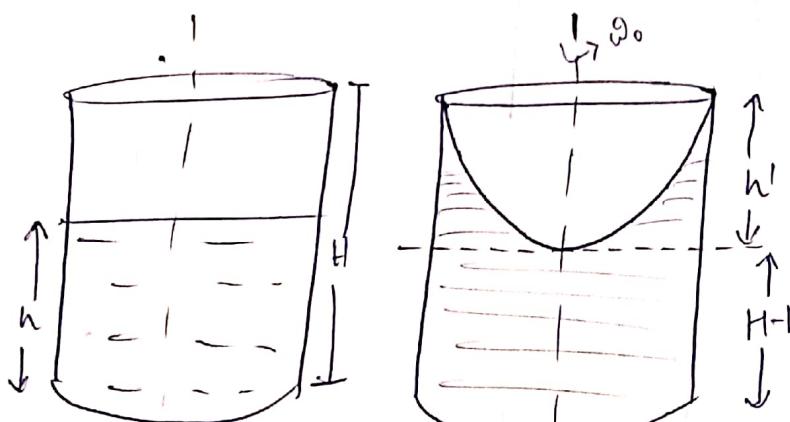
$$= \frac{\pi r^4 w^2}{4g}$$

$$= \frac{\pi r^2}{2} \cdot \left(\frac{r^2 w^2}{2g}\right) = \frac{\pi r^2 h}{2}$$



Volume of paraboloid is half of volume of cylinder in which it is present.

Spilling of liquid out of vessel



$$h' = \frac{R^2 w_0^2}{2g}$$

volume of liquid
is same, thus,

$$\pi R^2 h = \pi R^2 (H - h') + \frac{\pi R^2 h'}{2}$$

$$\Rightarrow h = H - \frac{h'}{2}$$

$$\Rightarrow h' = 2(H - h) = \frac{R^2 w_0^2}{2g}$$

$$\Rightarrow w_0 = \frac{2}{R} \sqrt{g(H-h)}$$

→ If

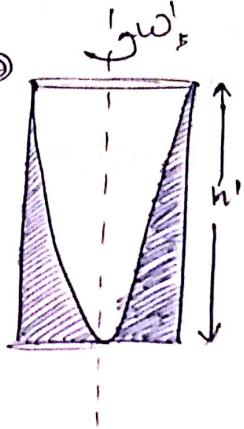
$$\omega > \omega_0$$

liquid will spill out of vessel.

→ If

$$\omega < \omega_0$$

liquid remains inside the vessel.



$$\omega' > \omega_0$$

$$H = \frac{\pi^2 \omega_0^2}{2g}$$

$$\omega' = \sqrt{\frac{2gH}{R^2}} = \frac{\sqrt{2gH}}{R}$$

$$\text{Volume of liquid} = \frac{\pi R^2 H}{2}$$

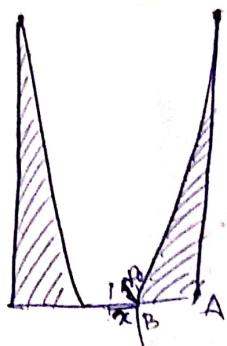
⇒ Initial amount of liquid $\pi R^2 h$ should be more than this.

$$\Rightarrow \pi R^2 h > \frac{\pi R^2 H}{2}$$

$$\Rightarrow h > \frac{H}{2}$$

Thus at this condition the bottom of paraboloid will touch the bottom of vessel.

⇒ Now on increasing ω' again bottom of vessel will be exposed to atmosphere



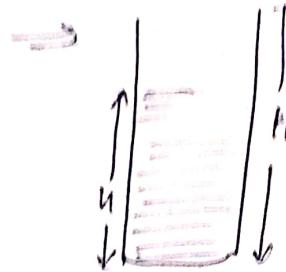
$$P_A - P_0 = \rho g H$$

$$P_A - P_0 = \frac{\rho \omega^2 \cdot (R^2 - x^2)}{2}$$

$$\rho g H = \frac{\rho \omega^2 \cdot (R^2 - x^2)}{2}$$

$$R^2 - \frac{2gH}{\omega^2} = x^2$$

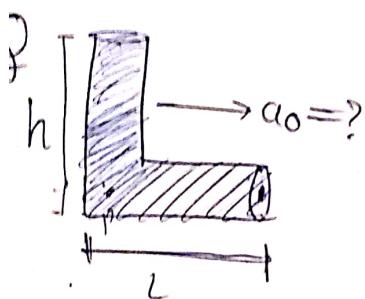
$$\text{Area of surface} = \pi x^2 = \pi \left(R^2 - \frac{2gH}{\omega^2} \right)$$



→ If $h > H_1/2$, spilling occurs first before exposure of bottom.

If $h = H_1/2$ both spilling and exposure ~~but~~ happens simultaneously.

If $h < H_1/2$ exposure occurs first and then spilling takes place.

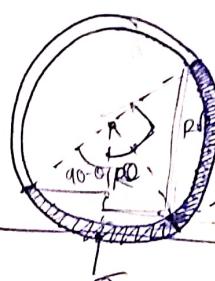


find acceleration required for liquid not to ~~fall~~ spill off from tube.

$$\text{Ans. } \rho a_0 \cdot l = \rho g h$$

$$\boxed{a_0 = gh/l}$$

Q



two of equal volume

~~$\rho g R (\alpha - \sin \alpha) = \rho g R v_2 + \sigma g R (1 - \cos \alpha)$~~
 ~~$\sigma g R (\cos \alpha - \sin \alpha) = \rho g R v_L$~~

$$\sin\left(\frac{\pi}{4} - \alpha\right) = \frac{2P}{\sigma}$$

$$\frac{\pi}{4} - \alpha = \sin^{-1}\left(\frac{2P}{\sigma}\right)$$

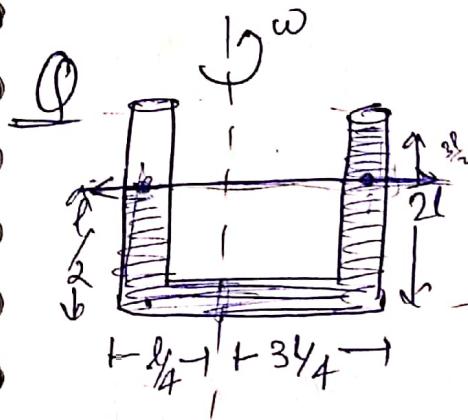
$$\frac{\pi}{4} - \sin^{-1}\left(\frac{2P}{\sigma}\right) = \sin \alpha$$

$$\sigma g R (\cos \theta - \sin \theta) = \rho g R (\sin \theta + \cos \theta)$$

$$(\sigma - \rho) \cos \theta = (\rho + \sigma) \sin \theta$$

$$\tan \theta = \frac{\sigma - \rho}{\rho + \sigma}$$

$$\theta = \tan^{-1} \left(\frac{\sigma - \rho}{\rho + \sigma} \right)$$

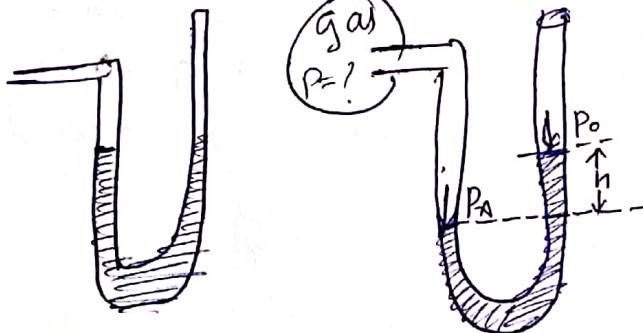


$$P_0 + \cancel{P_0} \quad \frac{\rho l^2 \omega^2}{32} = P_0 + \frac{9 \rho l^2 \omega^2}{32} + \frac{3 \rho g l}{2}$$

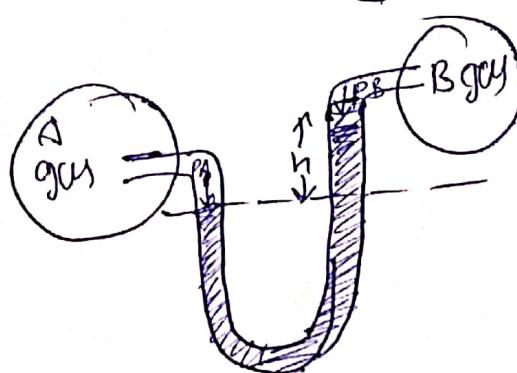
$$\frac{\rho l^2 \omega^2}{32} = \frac{3 \rho g l}{2}$$

$$\omega = \sqrt{\frac{6g}{l}}$$

Manometer



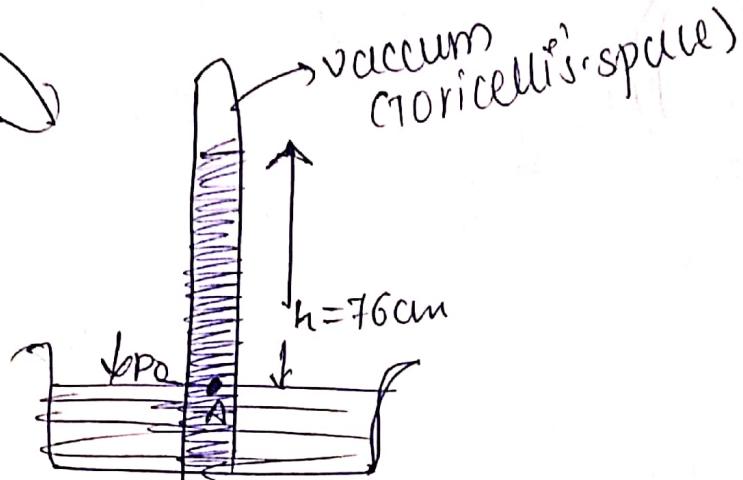
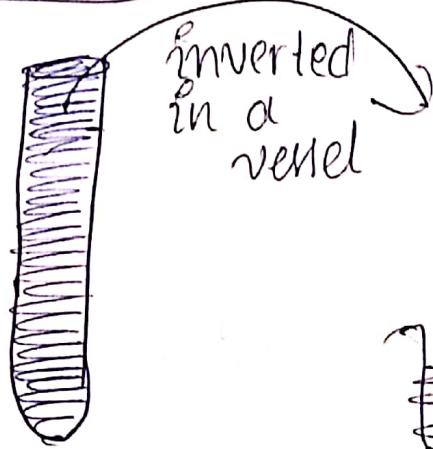
$$P_{\text{gas}} = P_0 + \rho_m \cdot g \cdot h$$



$$P_A = P_B + \rho_m g h$$

$$P_A - P_B = \rho_m g h$$

Barometer

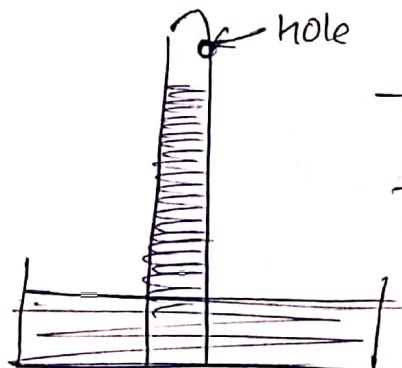


$$P_A = P_0 = \rho mg h$$

$$\Rightarrow P_0 = (13.6 \text{ gm/cc}) (980 \text{ cm/sec}^2)$$

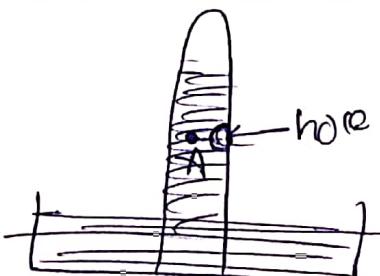
$$\boxed{P_0 = 1.013 \times 10^5 \text{ N/m}^2 \text{ (SI)}}$$

①



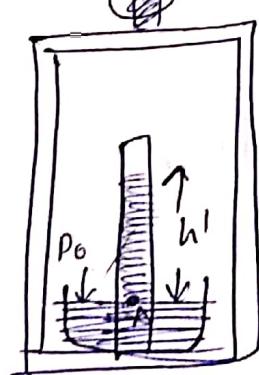
The total mercury column will fall down as value of pressure at top of column becomes P_0 .

②



$P_A < P_0 \Rightarrow$ Mercury won't come out and reading remains unchanged.

(3)



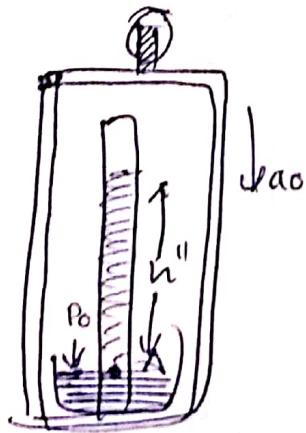
$$P_A = P_0$$

$$P_0 = \rho g h = \rho_m (g + a_0) h'$$

$$\Rightarrow h' = \frac{gh}{g + a_0}$$

$$\Rightarrow h' < h$$

(4)

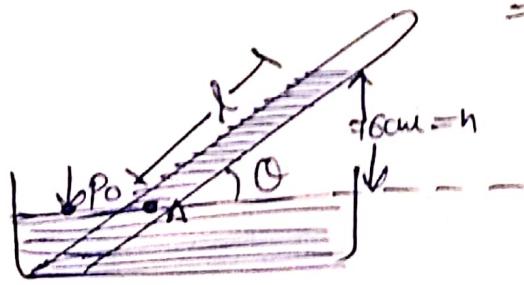


$$P_0 = P_A = \rho g h = \rho (g - a_0) h''$$

$$\Rightarrow h'' = \frac{gh}{g - a_0}$$

$$\Rightarrow h'' > h$$

(5)

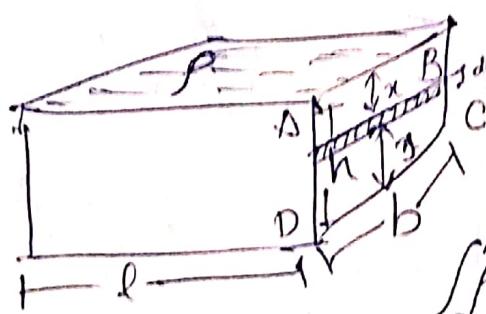


$$\sin \theta = h/l$$

$$\boxed{l = h / \sin \theta}$$

l = length of mercury column
in the tube.

\Rightarrow Force acting on a surface by fluid.



Neglect atmospheric pressure

find the force acting on ABCD?

Also find the torque acting on it
about axis CD?

$$\int_0^h \rho g x \cdot b dx = f$$

$$\rho g b \cdot \left[\frac{x^2}{2} \right]_0^h = F$$

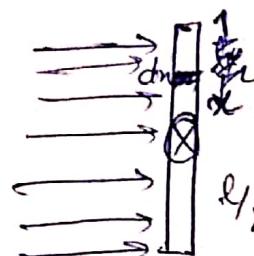
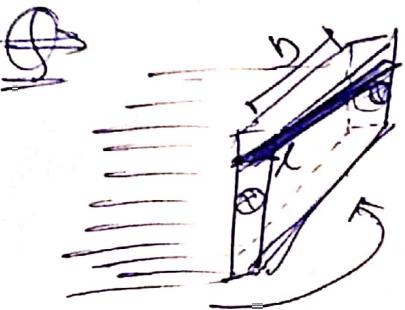
$$\boxed{F = \frac{\rho g b \cdot h^3}{2}}$$

$$dF = (dF) y$$

$$= b dy \cdot \rho g (h-y) \cdot y$$

$$= bd \left[\frac{hy^2}{2} - \frac{y^3}{3} \right]_0^h$$

$$= \rho g b \left[\frac{h^3}{2} - \frac{h^3}{3} \right]$$



$$d\tau_1 = \int_0^l \rho g b \left(\frac{l}{2} - x \right) dx \cdot b \cdot x$$

$$\tau_1 = \rho g b \left[\frac{\frac{l}{2}x^2}{2} - \frac{x^3}{3} \right]_0^{l/2}$$

$$= \rho g b \left[\frac{\frac{l^2}{4}}{2} - \frac{\frac{l^3}{8}}{3} \right]$$

$$= \rho g b \left[\frac{l^2}{8} \right]$$

$$\int d\tau_2 = \int \rho g \left(\frac{l}{2} + x \right) dx \cdot b$$

$$\tau_2 = \rho g b \left[\frac{l x}{2} + \frac{x^2}{2} \right]_0^{l/2}$$

$$= \frac{3 \rho g b l^3}{8}$$

$$T_{\text{ext}} = \frac{\rho g b l^2}{2} = F \cdot \frac{l}{2}$$

$$T_F = \frac{\rho g b l^3}{2}$$

$$\tau_1 = \int \rho g \left(\frac{l}{2} - x \right) dx \cdot b \cdot n$$

$$= \rho g b \cdot \left[\frac{\frac{l}{2}x^2}{2} - \frac{x^3}{3} \right]_0^{l/2}$$

$$= \rho g b \left[\frac{l^3}{16} - \frac{l^3}{24} \right]$$

$$= \frac{\rho g b l^3}{8} \times \frac{1}{6}$$

$$\tau_2 = \int \rho g \left(\frac{l}{2} + x \right) dx \cdot b \cdot x$$

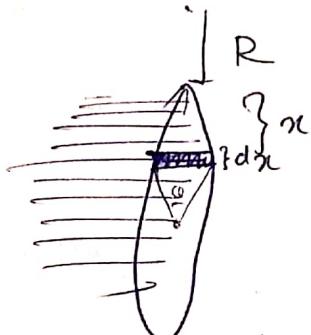
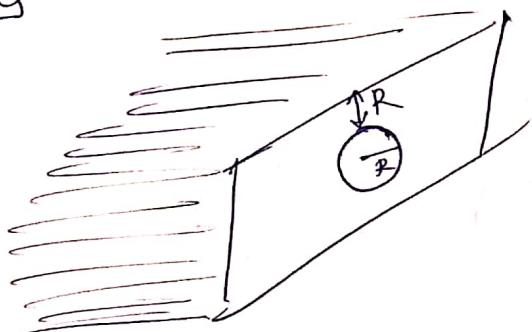
$$= \frac{\rho g b l^3}{8} \cdot \frac{5}{6}$$

$$T_{\text{net.}} = \frac{4 \rho g b l^3}{8 \cdot 6}$$

$$F \cdot \frac{l}{2} = \frac{\rho g b l^3}{12}$$

$$F = \frac{\rho g b l^3}{8}$$

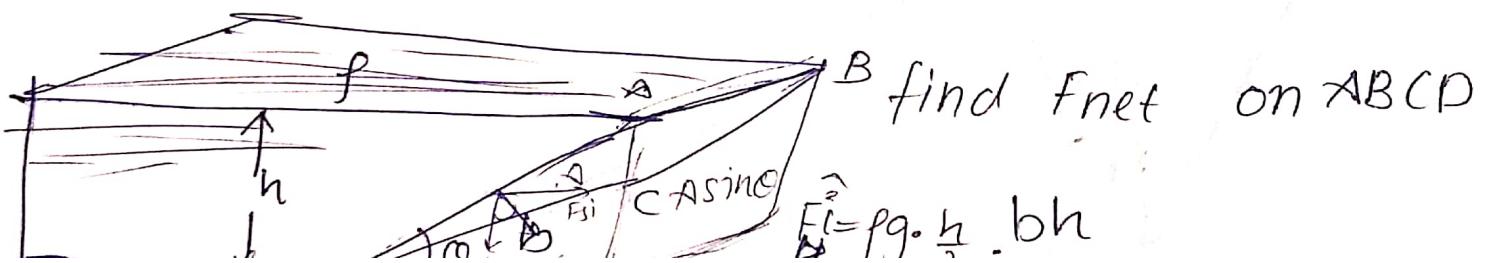
Q



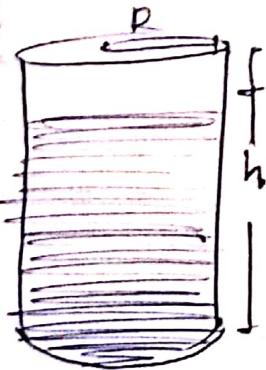
$$dF = \rho g (h+dh).$$

$$\begin{aligned}
 F_H &= \rho g \cdot (y_{cm}) \text{ (Area) (or to force)} \\
 &= \rho g (2R)(\pi R^2) \\
 &= 2\rho g \pi R^3
 \end{aligned}$$

Q



$$F = \rho g \cdot \frac{h}{2} \cdot bh$$



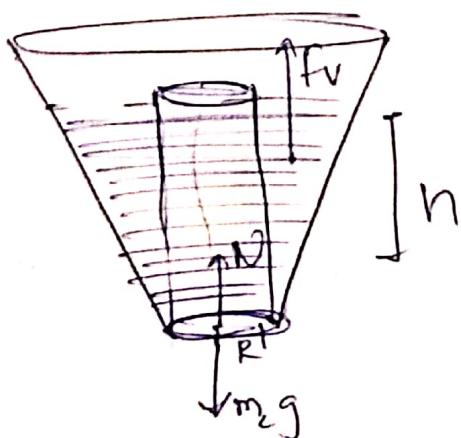
$$m_1 = \rho \pi R^2 h$$

$$F_{bottom} = \rho (\text{area})$$

$$= \rho (\pi R^2 h) g$$

$$= m_1 g$$

= weight of liquid above the bottom.



$$m_2 > m_1$$

$$F_{bottom} = \rho (\pi R^2 h) g$$

$$= m_1 g < m_2 g$$

$$N = m_1 g$$

$$F_v + N = m_2 g$$

$$F_v = (m_2 - m_1) g$$

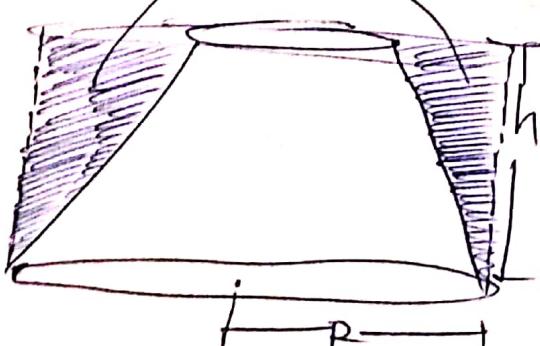
$$F_{bottom} = \rho g h \cdot \pi R^2$$

$$= m_1 g$$

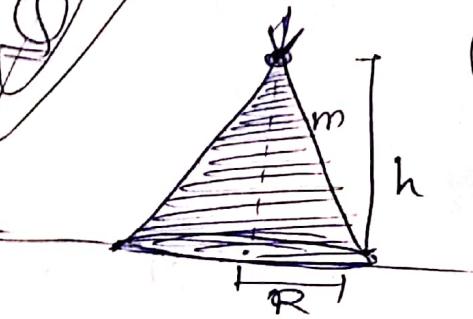
$$m_3 g < m_1 g$$

$$m_3 g + F_w = m_1 g$$

$$F_w = (m_1 - m_3) g$$

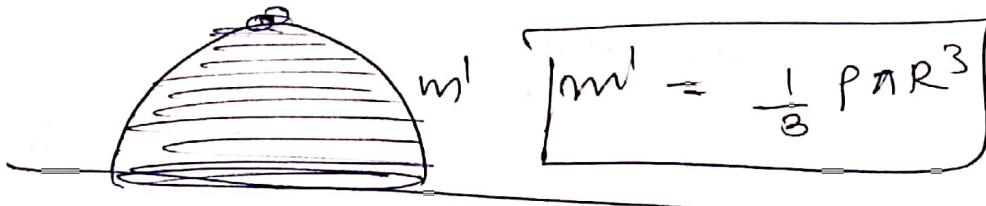


Force ~~exerted~~ by walls on liquid inside vessel is weight of imaginary liquid remained after make arrangement beside



$$(\rho \pi R^2 h - \frac{\rho \pi R^2 h}{3}) g \geq m g$$

$$\frac{2 \rho \pi R^2 h}{3} \geq m g$$



$$m' = \frac{1}{3} \rho \pi R^3$$

- Force of buoyancy :-

~~$F_B = \rho g \pi R^2 h$~~

$F_A = (\rho g x_0 + P_0) \pi R^2$

$F_B = (\rho g m_0 + P_0) \pi R^2$

$F_B > F_A$

$\rightarrow \text{Net force} = F_B - F_A$

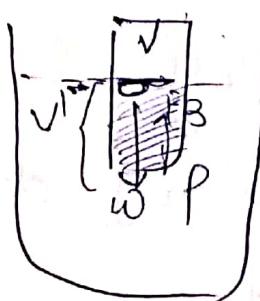
$= \rho g h \pi R^2$

$= (\rho g) \text{Volume of cylinder}$

F_B = weight of liquid displaced by object.

$F_B = (\text{volume of liquid displaced}) \cdot \text{f liquid} \cdot g$

$F_B = (\text{volume of object immersed}) \cdot \text{f liquid} \cdot g$



$\sigma V g = \rho V' g$

$V' = \frac{\sigma V}{\rho}$

if $\sigma < \rho$; $\frac{\sigma}{\rho} < 1$, $\frac{V'}{V} < 1$ (floatation is possible)

floats with partially submersion.

If $\sigma = \rho$, $\frac{\sigma}{\rho} = 1 \Rightarrow \frac{V'}{V} = 1$ (floatation is possible)
complete submersion.

If $\sigma > \rho$, $\frac{\sigma}{\rho} > 1 \Rightarrow \frac{V'}{V} > 1 \Rightarrow V' > V$ (body sinks).
not possible

Q



$$\rho V' g - \sigma V g = \sigma V a$$

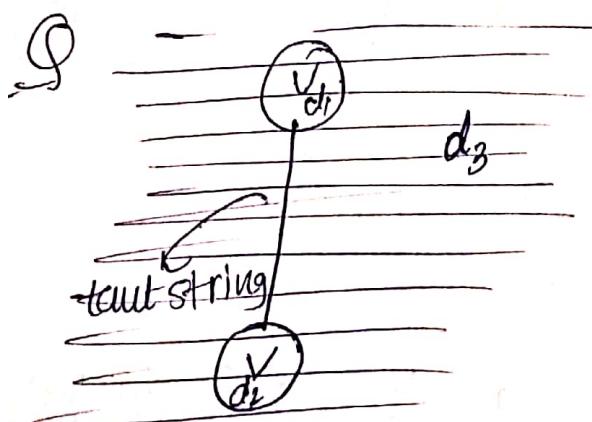
$$\frac{(\rho - \sigma) g}{\sigma} = a$$

$$V = \sqrt{2 \times \frac{\rho - \sigma}{\sigma} g \times d}$$

$$\frac{1}{2} m v^2 = mgh$$

$$\frac{1}{2} \times \frac{\sigma(\rho - \sigma)}{\sigma} g d = mgh$$

$$h = \frac{d(\rho - \sigma)}{\sigma}$$



$$V d_1 g + T = V d_3 g$$

$$T + V d_2 g = V d_2 g$$

$$V d_3 g - V d_1 g = V d_2 g - d_3 V g$$

$$2 d_3 = d_1 + d_2$$

$$d_3 = \frac{d_1 + d_2}{2}$$

$$F_B = \rho g A_1 \quad | \quad \rho = \text{constant} \\ \Rightarrow \rho g A_1 \quad | \quad \Rightarrow \rho g A_1 d h \\ \Rightarrow \rho g A_1 d h$$

If 6-surface plots (area) of flow is arranged in form of curve, if density of water is a form of flow then minimum value of flow

$$\rho g A_1 d h = \rho g A_1 h$$

$$\rho g A_1 d h = \rho g A_1 h$$

$$\rho g A_1 d h = \rho g A_1 h$$

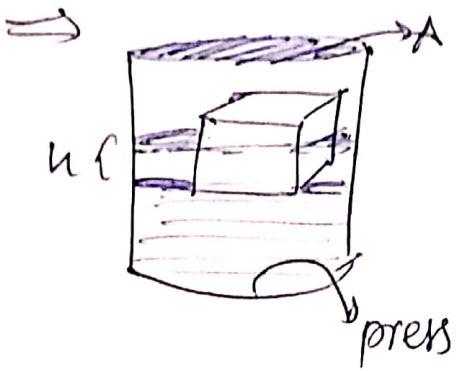
$$f_2 = \frac{[\text{volume of trapezoid}]}{[\text{area of trapezoid}]} (F_B) (C_g + C_s)$$

C_g = forward C_s = stern

$$(C_g + C_s) = 2 (C_g + C_s) / 2$$

$$TVL = \frac{dV}{dL}$$

area = $\frac{1}{2} (a_1 + a_2) h$



$$m g = P \cdot L^2 \cdot x g$$

$$x = \frac{m}{PL^2}$$

$$\pi x L^2 = \frac{m}{P} = \cancel{A} (A - L^2) h$$

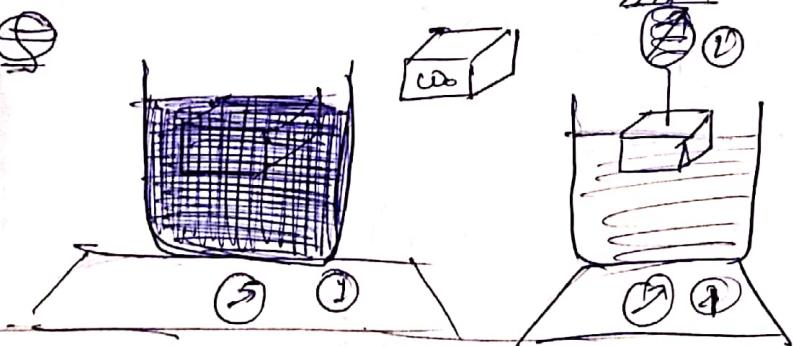
$$h = \frac{m}{P(A - L^2)}$$

increase in pressure at bottom.

$$\Delta P = \rho_w g h$$

$$\Delta P = \rho_w g \cdot \frac{m}{h \cancel{L} (A - L^2)}$$

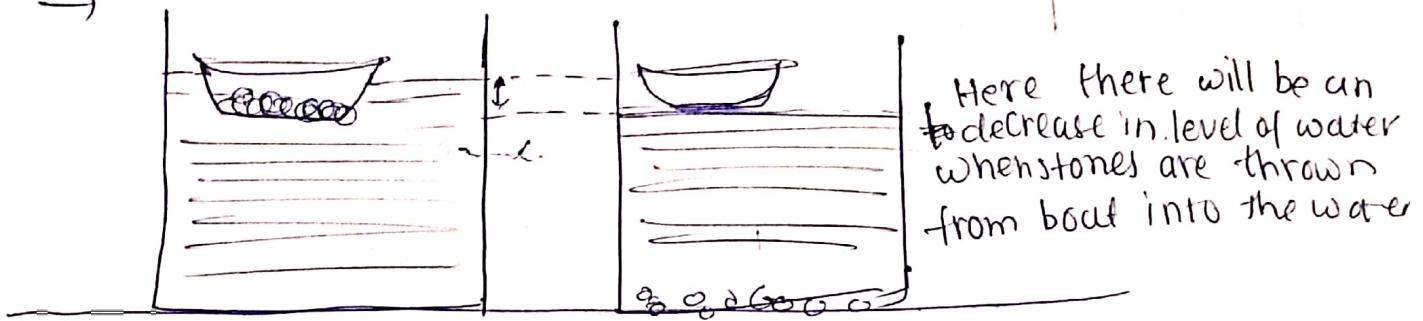
$$\Delta P = \frac{mg}{A - L^2}$$



Reading ① = wt. of water + beaker

Reading ② = $w_0 + F_B$
= apparent wt.

Reading ③ = $w_0 + (water + beaker) + F_B$.

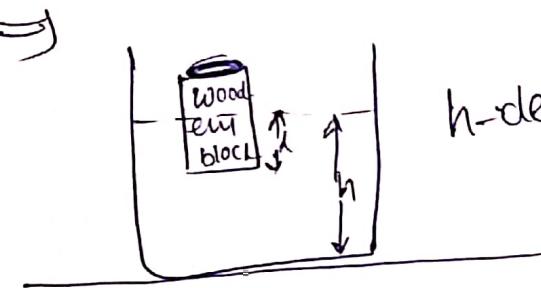


$$F_B = w$$

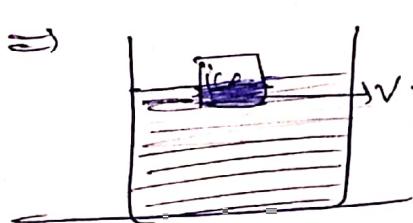
$$V_m \cdot \rho_w \cdot g = \sigma v g$$

$$V_m = \frac{V \cdot \sigma}{\rho_w}$$

if $\sigma > \rho_w$, object sinks



h -decreases as well as l -decreases



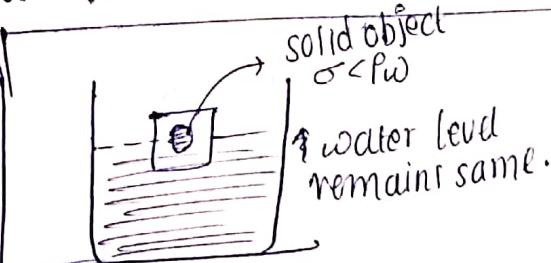
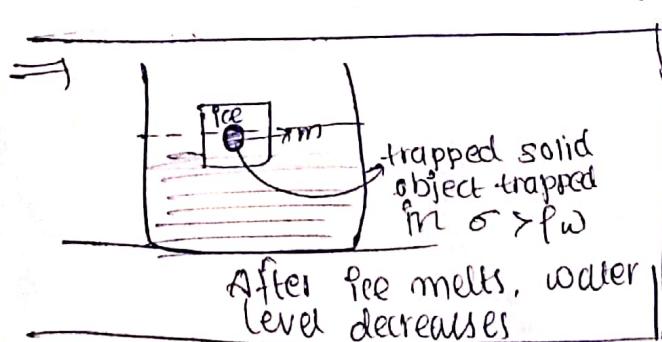
if ice melts completely,
volume produced = V_1

$$m = V_{\text{ice}} \rho_{\text{ice}} = V_1 \rho_w$$

$$V_1 = \frac{m}{\rho_w}$$

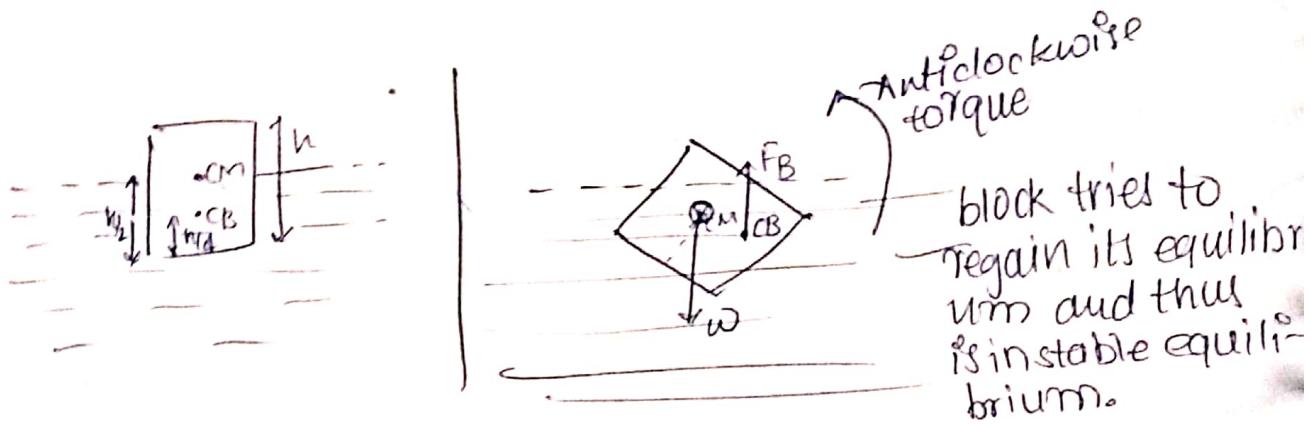
$$V_2 = \frac{m}{\rho_w}$$

Thus water level remains same.

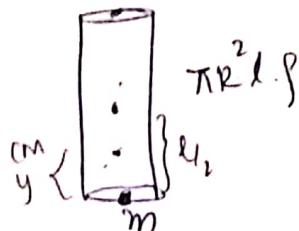


Centre of buoyancy

- ⇒ point where force of buoyancy will act.
- ⇒ centre of immersed volume.



(33)



$$\pi R^2 l \rho$$

$$\left[\frac{\pi R^2 l \rho}{m + \pi R^2 l \rho} \cdot \frac{l}{2} \right] = y_{cm}$$

$$(m + \pi R^2 l \rho) g = \pi R^2 \cdot x \cdot \sigma \cdot g$$

$$\frac{\pi R^2 l \rho \cdot \frac{l}{2}}{m + \pi R^2 l \rho} = \frac{m + \pi R^2 l \rho}{\pi R^2} = \frac{x}{2}$$

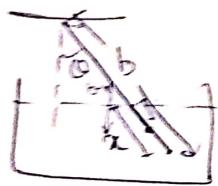
$$\sigma \cdot (\pi R^2)^2 \cdot \frac{l^2}{4} = (m + \pi R^2 l \rho)$$

$$\pi R^2 l \sqrt{\sigma \rho} = m + \pi R^2 l \rho$$

$$\pi R^2 l (\sqrt{\sigma \rho} - \rho) = m$$

$$\pi R^2 l \rho \left[\sqrt{\frac{\sigma}{\rho}} - 1 \right] = m$$

(18)



$$P. \frac{F}{2} \times \frac{b}{2} = \left(\frac{m. b. f. x. g}{18} \right) \times \left(\frac{b+x}{2} \right)$$

$$\frac{5}{9} \times \frac{b^2}{2} = \left(\frac{2b-x}{2} \right) x$$

$$5b^2 = 18bx - 9x^2$$

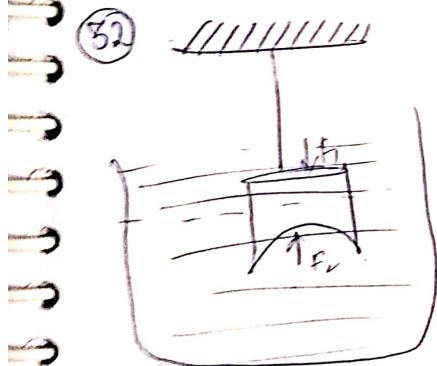
$$9x^2 - 18bx + 5b^2 = 0$$

$$9x^2 - 15bx - 3bx + 5b^2 = 0$$

$$3x[3x-5b] - b[3x-5b] = 0$$

$$x = \frac{b}{3}$$

(32)

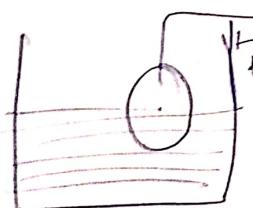


$$F_2 - F_1 = F_B = \rho g V$$

$$F_2 = \rho g V + \rho g h \pi R^2$$

$$= \rho g (\pi R^2 h + V)$$

(51)



$$F = \frac{V}{2} \left(\rho_{air} g - \rho_{water} \frac{V}{2} g \right) x + \frac{mg}{L} \frac{2x^2}{2}$$

$$Vg \left[2.7 \times 10^3 - 0.5 \times 10^3 \right] x + \frac{mgx^2}{2} = \frac{mg \left(\frac{2x^2}{2} \right)}{L}$$

$$2Vg \left[2.2 \times 10^3 \right] x + \frac{mgx^2}{2} = \frac{mg}{L} \left(1 - \frac{x^2}{L} \right)$$

$$\text{Now } \left(\rho_{air} - \frac{\rho_{water}}{2} \right) x = \frac{mg}{L} \times \left(\frac{y-x}{2} \right)$$

$$\frac{V}{2} \left[2.7 \times 10^3 \right] = \frac{4.4 \times 10^3}{2} \times \left[\frac{y-x}{2} \right]$$

$$\frac{4}{3} \left(\frac{22}{7} \right) \times 12.5 \times 10^6 \times \frac{10^6}{10^9} \times \frac{10^3}{x} = 500 \times 11$$

(28)

$$\begin{aligned}\omega_1 &= \rho V g - d_1 V g \\ \omega_1 &= (\rho - d_1) V g \\ \hline \omega_1 &= (\rho - d_2) V g\end{aligned}$$

$$\omega_3 = (\rho - d_3) V g$$

$$\omega =$$

$$\omega_1 - \omega_2 = (d_2 - d_1) V g$$

$$\left(\frac{\omega_1 - \omega_2}{d_2 - d_1} \right) \frac{l}{g} = V$$

$$\omega_1 (\rho - d_1) = \omega_2 (\rho - d_1)$$

$$\cancel{\omega_1} (\omega_1 - \omega_2) \rho = \omega_1 d_2 - \omega_2 d_1$$

$$\rho = \frac{\omega_1 d_2 - \omega_2 d_1}{\omega_1 - \omega_2}$$

Parcels law

If at any point of fluid pressure increase, that increase is distributed equally to every point without diminishing its value



$$\begin{aligned}P_A &= P_0 + F/A \\ P_B &= P_0 + F/A + \rho g h\end{aligned}$$

fluid Dynamics

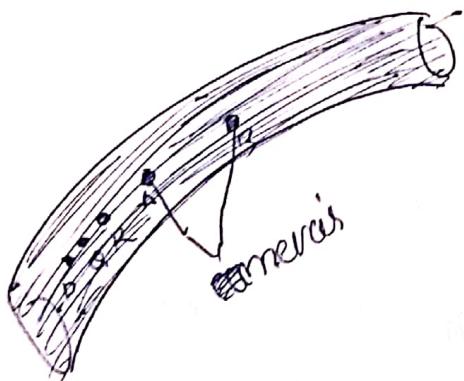
→ fluid is ideal. which means :-

- non compressible liquid
- non-viscous fluid
- irrotational
(fluid doesn't acquire any angular velocity at any point of fluid flow.)

Ideal fluid flow

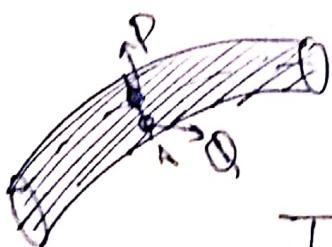
1. Laminar

Stream line flow



if all particles are having same velocity (dir, mag) at a point in the fluid flow then flow is called stream line flow. velocity at A may be different from that at B. But at a point all particles must have same velocity at a given point.

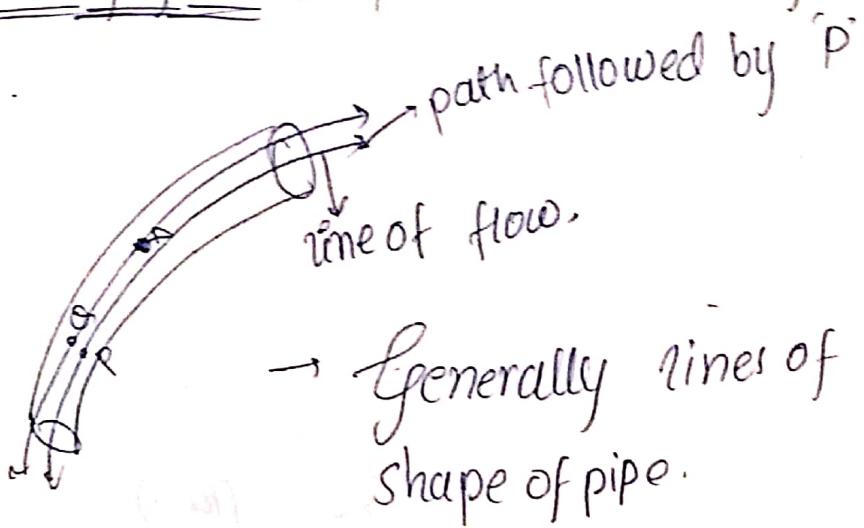
Turbulent flow



At a point in fluid flow different particles will have different velocities. The flow is turbulent one.

$$[V_{P\alpha A} \neq V_{Q\alpha B}]$$

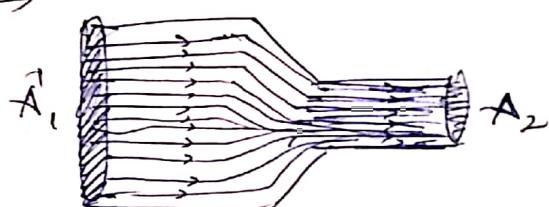
\Rightarrow line of flow = Path followed by a particle in fluid flow.



\rightarrow Generally lines of flow will be parallel to shape of pipe.

\rightarrow At any point if we draw a tangent it'll give us direction of velocity at that point.

\rightarrow If lines of flow are parallel then it's streamline flow otherwise turbulent.



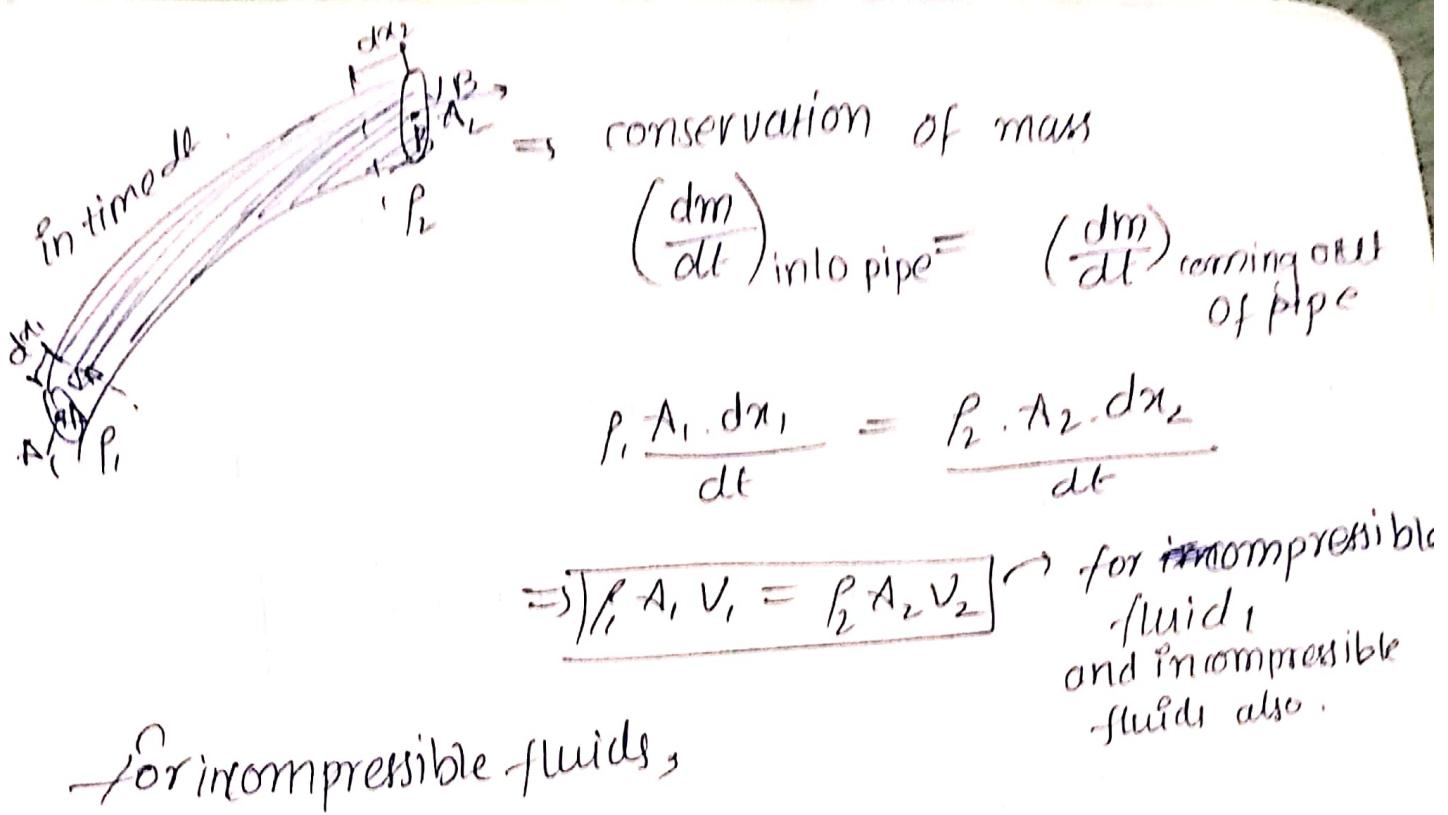
\Rightarrow line of flow density

= lines of flow per unit area.

\Rightarrow line flow density \propto velocity of that

\Rightarrow Ideal fluid always follows $\frac{\text{point-}}{\text{area}}$ Equation of continuity. (

\rightarrow Rate of mass of liquid entering the pipe is same as rate of mass of fluid leaving the pipe.



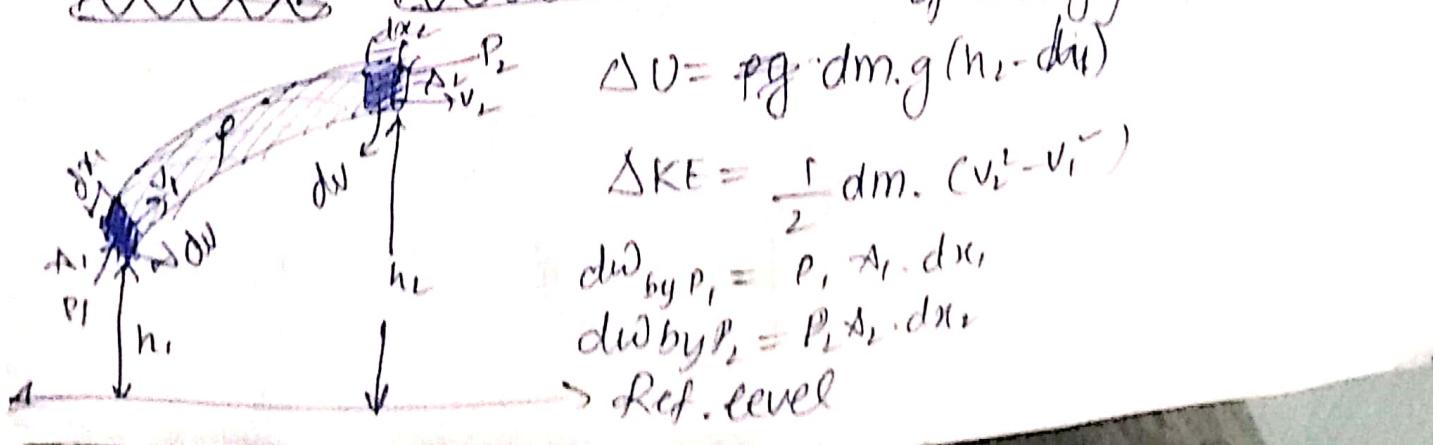
For incompressible fluids,

$$\Rightarrow \rho_1 = \rho_2$$

$$\Rightarrow A_1 V_1 = V_2 A_2 \Rightarrow \underbrace{AV}_{\text{constant rate of volume flow}}$$

- ⇒ Incompressible fluids have same rate of volume as well as mass flow at every
- ⇒ But compressible fluid rates of volume and mass flow are not equal.

Bernoulli's Theorem (consequence of conservation of energy)



WE theorem says:

$$\text{Work done} = \Delta KE$$

$$\Rightarrow (P_1 dA dx_1 - P_2 dA dx_2) - dm g (h_2 - h_1) = \frac{1}{2} dm (V_2^2 - V_1^2)$$

$$\Rightarrow (P_1 dV - P_2 dV) - \rho dV \cdot g (h_2 - h_1) = \frac{1}{2} \rho dV (V_2^2 - V_1^2)$$

$$\Rightarrow \boxed{P_1 + \rho gh_1 + \frac{1}{2} \rho V_1^2 = P_2 + \rho gh_2 + \frac{1}{2} \rho V_2^2}$$

$$\Rightarrow \boxed{P + \rho gh + \frac{1}{2} \rho V^2 = \text{constant}}$$

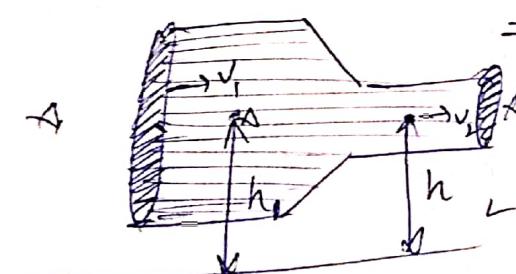
$$\Rightarrow \boxed{\frac{P}{\rho} + gh + \frac{V^2}{2} = \text{constant}}$$

P/ρ = pressure head

gh = 'U' head

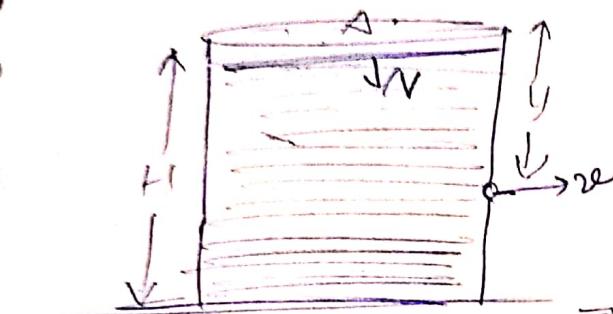
$\frac{V^2}{2}$ = KE head

② Bernoulli's Theorem along horizontal level in fluid flow.


$$\Rightarrow A_1 V_1 = A_2 V_2$$
$$\Rightarrow P_1 + \rho g h_1 + \frac{1}{2} \rho V_1^2 = P_2 + \rho g h_2 + \frac{1}{2} \rho V_2^2$$
$$\Rightarrow \boxed{P_1 + \frac{1}{2} \rho V_1^2 = P_2 + \frac{1}{2} \rho V_2^2}$$

Application of BT and EC :-

② Velocity of Efflux.



A \Rightarrow Area of cross section of vessel

a \Rightarrow area of cross section of $\approx 10^{-6}$

$a \ll A$

\Rightarrow By EC,

$$AV = a, v$$

$$\Rightarrow v = \frac{(av)}{A}$$

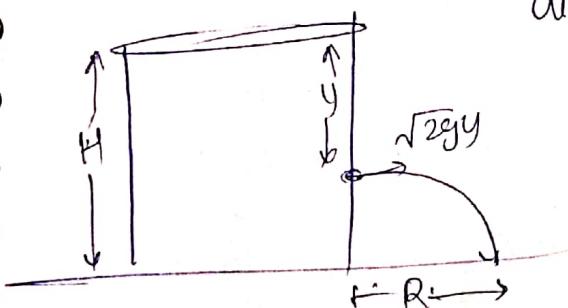
$$TV \ll v$$

$$\text{By BT, } P_0 + \frac{1}{2} PV^2 + PGH = P_0 + \frac{1}{2} PV^2 + PG(H-y)$$

$$PGy = \frac{1}{2} P(V^2 - v^2)$$

$$\Rightarrow v = \sqrt{\frac{2gy}{1 - a^2/A^2}} \quad \text{as } a \ll A \Rightarrow v = \sqrt{2gy}$$

similar to velocity acquired by a part by freely falling body through a distance y .



$$v = \sqrt{2gy} \cdot \sqrt{\frac{2(H-y)}{g}}$$

$$= 2\sqrt{y(H-y)}$$

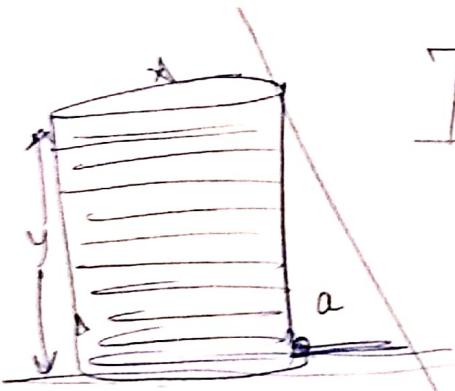
$$v^2 = 4y(H-y)$$

$$2r \left(\frac{dv}{dt} \right) = 4y(-1) + 4(H-y) = 0$$

$$4H - 8y = 0 \Rightarrow y = H/2$$

\Rightarrow thus R will be maximum at $y = H/2$.

$$TR_{\max} = H$$



$$A(a_r) = a \cdot (a_w)$$

$$v = \sqrt{\frac{2gy}{1 - a^2/A}}$$

$$\frac{dv}{dt} = \left(\sqrt{\frac{2gy}{1 - a^2/A}} \right) \left(\frac{1}{\rho V g} \right) \cdot \frac{a^2 g}{1 - a^2/A}$$

\Rightarrow

$$a_w = \frac{g}{1 - a^2/A} \approx g$$

$$a(a_r) \approx a \cdot g$$

$$TR \approx ag/A$$

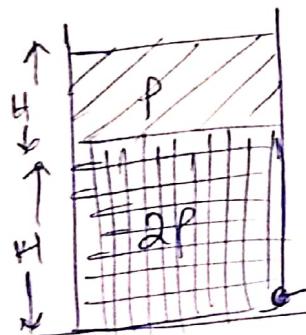
$$A \cdot v = a \cdot \sqrt{2g} y$$

$$A \cdot \frac{dv}{dt} = a \cdot \frac{\sqrt{2g}}{2\sqrt{y}} \cdot \frac{dy}{dt}$$

$$A \cdot \frac{dv}{dt} = a \frac{\sqrt{2g}}{2\sqrt{y}} \frac{a\sqrt{2g}y}{A}$$

$$\frac{dv}{dt} = \left(\frac{a^2}{A^2} g \right)$$

Q



~~$$p_0 + \frac{1}{2} \rho v^2 + \rho g(2H) = p_0 + \frac{1}{2} \cdot 2\rho v^2 + \frac{2\rho}{2}$$~~

~~$$2\rho gH = \frac{1}{2} 2\rho v^2 - \frac{1}{2} \rho v^2$$~~

~~$$= \rho \left[v^2 - \frac{a^2 v^2}{2A^2} \right]$$~~

~~$$2\rho gH = \rho \left[v^2 \right] \left[1 - \frac{a^2}{2A^2} \right]$$~~

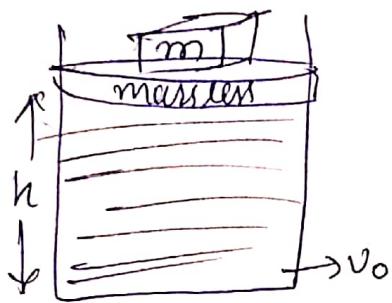
$$\sqrt{\frac{2gH}{1 - \frac{a^2}{2A^2}}} = v$$

$$p_0 + \frac{1}{2} 2\rho v^2 + \rho gH = p_0 + 2\rho gH + \frac{1}{2} 2\rho v^2$$

$$\rho v^2 = 3\rho gH$$

$$Tr = \sqrt{3gh}$$

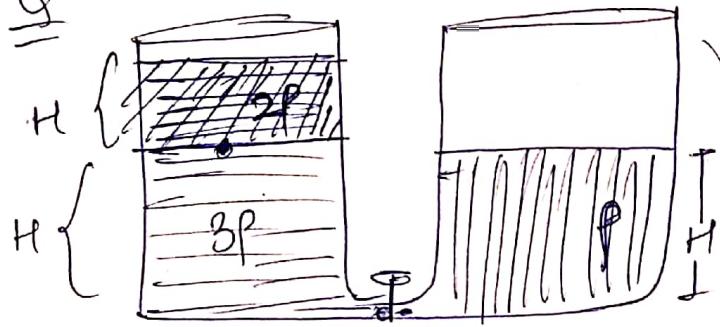
Q



$$\rho_0 + \left(\frac{mg}{A}\right) + \frac{1}{2} \rho v^2 + \rho gh = \rho_0 + \frac{1}{2} \rho v_0^2 + 0$$

$$\sqrt{2 \left(\frac{mg}{\rho A} + \rho gh \right)} = v_0$$

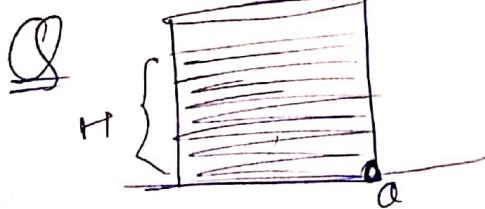
Q



$$\rho_0 + 2\rho gh + \frac{1}{2} (2\rho) v^2 = \rho_0 + \rho gh + \frac{1}{2} (3\rho) v^2 + 3\rho gh$$

$$4\rho gh = \frac{1}{2} 3\rho v^2$$

$$\frac{8gh}{3} = v^2$$



time taken to empty is $\propto A$
 $\propto H$
 $\propto 1/a$

By, EC

$$A \cdot -\frac{dy}{dt} = a \cdot \sqrt{2g} \cdot \sqrt{y}$$

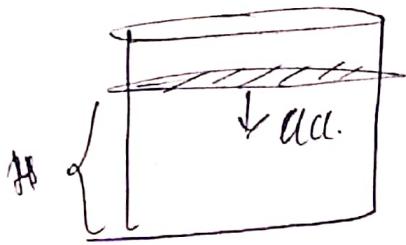
$$A \cdot \int_{H}^0 \frac{dy}{\sqrt{y}} = a \sqrt{2g} \int dt$$

$$A [2\sqrt{y}]_H^0 = - (a\sqrt{2g}) t$$

$$A [2\sqrt{t}] = a\sqrt{2g} (t) \Rightarrow t = \frac{A}{a} \sqrt{\frac{2H}{g}}$$

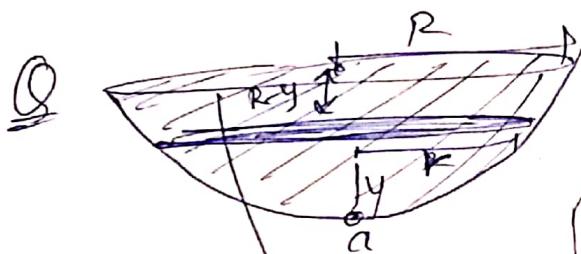
In general to empty tank from H_1 to H_2

$$t = \frac{A}{a} \sqrt{\frac{2}{g}} (H_1 - H_2) \quad (H_1 > H_2)$$



$$H = \frac{1}{2} a t^2$$

$$\frac{2H}{a^2 \cdot g} = t^2 \Rightarrow t = \frac{A}{a} \sqrt{\frac{2H}{g}}$$



$$A \cdot V = a \cdot \vartheta$$

$$P_0 + \rho g R + \frac{1}{2} \rho v^2 = \frac{1}{2} \rho v^2 + P_f$$

$$v = \sqrt{2gR}$$

$$A \cdot \frac{dr}{dt} = a \cdot \sqrt{2gr}$$

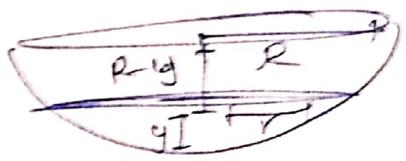
$$\frac{A}{a \sqrt{2g}} \cdot [2\sqrt{r}]_R^0 = t$$

$$A \cdot V = a \vartheta$$

$$\frac{dA}{dt} V + A \cdot \frac{dV}{dt} = a \cdot \frac{d}{dt} (\sqrt{2gr})$$

$$\frac{d}{dt} (\pi R^2) + A \frac{dV}{dt} = a \cdot \frac{d}{dt} (\sqrt{2gr})$$

$$2\pi R \left(\frac{dr}{dt} \right) + A (a \vartheta) = a \sqrt{2g} \cdot \frac{1}{2\sqrt{r}} \cdot \frac{dr}{dt}$$



$$r^2 = R^2 + y^2 - 2Ry$$
$$R^2 - y^2 = r^2 - 2Ry$$

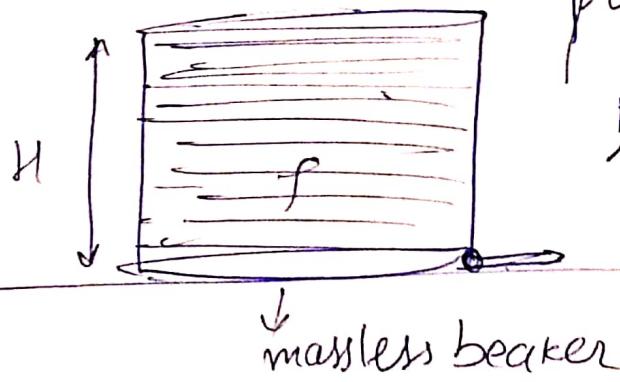
$$\int_R^0 \pi(2Ry - y^2) \frac{dy}{dt} = -a \cdot \sqrt{2g} \cdot \sqrt{y} \int_0^t dt$$

$$\int_R^0 \pi(2R\sqrt{y} - y^{3/2}) dy = a\sqrt{2g}(t)$$

$$\left[\frac{\pi(2R)(y)^{3/2}}{3/2} - \frac{\pi y^{5/2}}{5} \right]_R^0 = -a\sqrt{2g}(t)$$

$$- \frac{4R\pi}{3} \cdot R^{3/2} + \frac{2\pi}{5} \cdot R^{5/2}$$

Force due to liquid flow



$$\rho a \cdot \left(\sqrt{\frac{2gH}{1-\alpha}} \right)^2 = \rho AH \cdot [acc]$$

$$[acc] = \frac{2gH\alpha}{a^2 - \alpha^2}$$

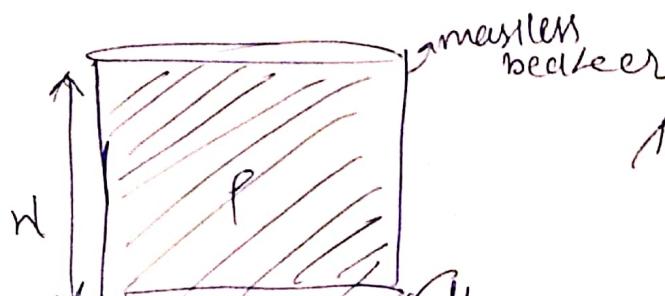
$$[acc] \approx \frac{2gH\alpha}{HA} = \frac{2ga}{A}$$

Independent of height of fluid.

A beaker with mass,

$$\rho a \left(\frac{2gH}{a^2 - \alpha^2} \right)^2 = (\rho AHg + m) a$$

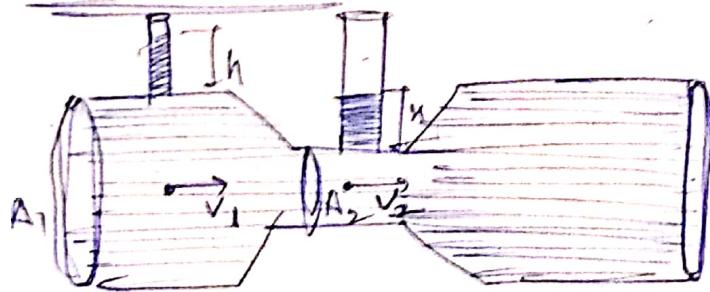
$$a = \frac{2\rho g ab}{\rho AHg + m}$$



$$\rho a \cdot 2gy = \mu \rho AH g$$

$$Ty \Rightarrow \frac{\mu AH}{2a}$$

VENTURI METER :-



→ A device to measure velocity of flow.

By E.C.

$$A_1 V_1 = A_2 V_2$$

By B.T,

$$P_0 + \rho g(h+x) + \frac{1}{2} \rho V_1^2 = P_0 + \rho g x + \frac{1}{2} \rho V_2^2$$

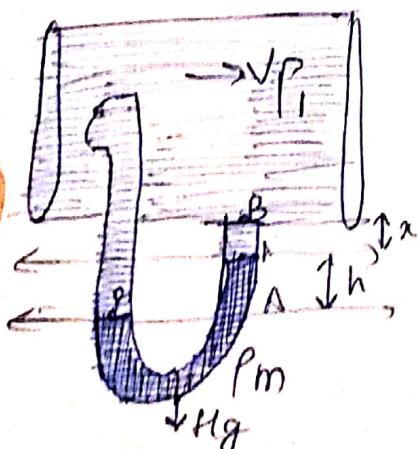
$$\rho g h = \frac{1}{2} \rho \left(\frac{A_1^2}{A_2^2} - 1 \right) V_1^2$$

$$V_1 = A_2 \sqrt{\frac{2gh}{A_1^2 - A_2^2}}$$

$$V_2 = A_1 \sqrt{\frac{2gh}{A_1^2 - A_2^2}}$$

$$\text{Rate flow} = A_1 A_2 \sqrt{\frac{2gh}{A_1^2 - A_2^2}}$$

pilot tube :- A device used to measure speed of fluid



$$P_A - P_B = \rho_1 g x + \rho_m g h$$

$$T E_A = T F_B$$

$$P_A + \rho_1 x + \rho_1 = P_B + \frac{1}{2} \rho_1 V^2 + \rho_1 g (h+x)$$

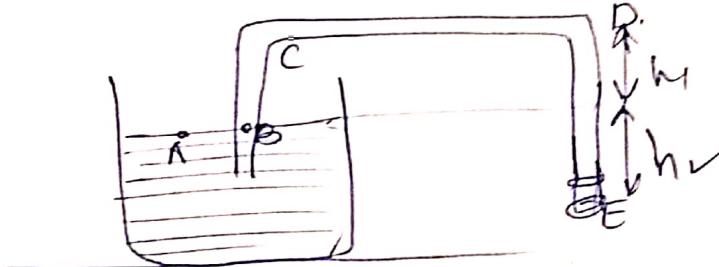
$$\rho_1 g x + \rho_m g h = \frac{1}{2} \rho_1 V^2 + \rho_1 g (h+x)$$

$$\Rightarrow \sqrt{\frac{2(P_m - P_1)gh}{P_1}} = v$$

$$\text{if } P_1 \ll P_m \Rightarrow v = \sqrt{\frac{2f_m gh}{P_1}}$$

\Rightarrow Siphon tube:-

$$TE_E = TED$$



$$P_0 - \rho gh_2 + \frac{1}{2} \rho v^2 = P_D + \frac{1}{2} \rho v^2 + \rho gh_1$$

$$P_D = P_0 - \rho g(h_1 + h_2)$$

$$P_C = P_D$$

$$TE_B = TED$$

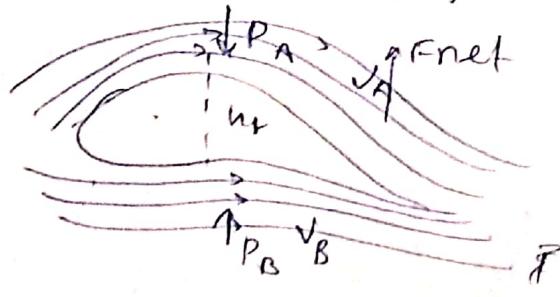
Note :- To make siphon tube work end of tube should be below fluid level.

$$P_B = P_C + \rho gh_1$$

$$\Rightarrow P_B = P_0 - \rho gh_2$$

Condition for working of siphon tube is $h_2 > 0$.

\Rightarrow Aerodynamic lift



$$TE_A = TED$$

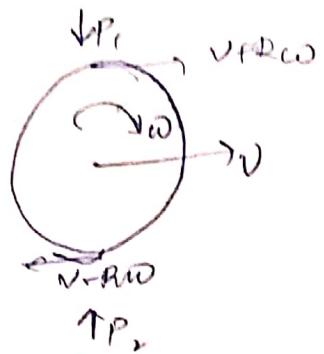
$$P_A + \frac{1}{2} \rho v_A^2 = P_B + \frac{1}{2} \rho v_B^2 + \sigma$$

$$\Rightarrow P_A - P_B = \frac{1}{2} \rho (v_2^2 - v_1^2)$$

$$\Rightarrow F_{net} = (P_A - P_B) \frac{\text{Area of wing}}{2}$$

$$= \frac{1}{2} \rho C (v_2^2 - v_1^2) (\text{Area of wing})$$

⇒ Magnus Effect



$$P_2 + \frac{1}{2} \rho (V + R\omega)^2$$

$$= P_1 + \frac{1}{2} \rho (V - R\omega)^2$$

$$\Rightarrow \rho g (2R)$$

$$(f_{\text{net}})_{\text{upward}} = (P_2 - P_1) (\text{Area effective})$$

$$= (P_2 - P_1) \pi R^2$$

Q If velocity of particle at point in stream flow is $\vec{v} = (V_0 + bx)\hat{i} + by\hat{j}$, then the eqn of streamline is?

$$\Rightarrow V_x = V_0 + bx = \frac{dx}{dt}$$

$$V_y = -by = \frac{dy}{dt}$$

$$\frac{dy}{dx} = \frac{-by}{V_0 + bx}$$

$$\int \frac{dy}{y} = -b \int \frac{dx}{V_0 + bx}$$

$$by = -b \ln(V_0 + bx)$$

$$y = \frac{1}{b} \ln(V_0 + bx)$$