

ELASTICITY: (on app forces, solid bodies deform)
 ↓
 Non-rigid bodies

Elastic Object:

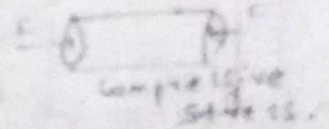
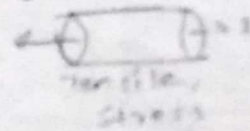
→ If object is able to regain its shape & size after removal of deforming forces object is said to be elastic.
 ↳ external forces.

Inelastic Object: Plastic Object:

→ If object is unable to regain its shape & size after removal of deforming forces object is said to be plastic.

STRESS: defined as force acting per unit area.

$$\text{Stress} = \frac{\text{Force}}{\text{Area}}$$



STRAIN:

$$\text{Strain} = \frac{\text{change in dimension}}{\text{Original dimension}}$$

Stress \propto strain (generally but not always)

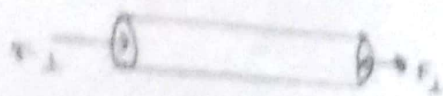
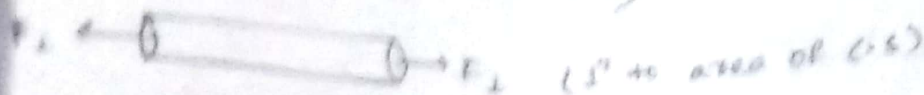
$$\text{Stress} = [\text{Modulus}] \text{Strain}$$

$$\Rightarrow \boxed{\frac{\text{Stress}}{\text{Strain}} = [\text{Modulus}]}$$

(only for small deformation)

→ For large deformation $\frac{\text{Stress}}{\text{Strain}} \downarrow$

Young's Modulus (length variation)



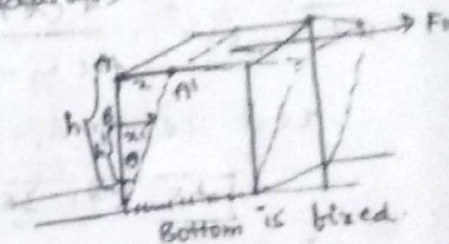
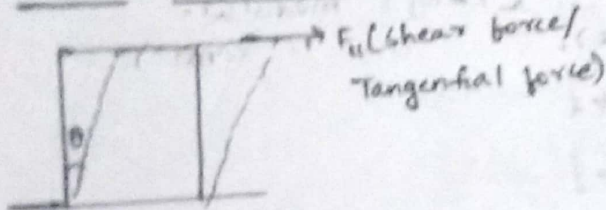
Longitudinal stress = $\frac{F_1 \text{ (or } F_2 \text{), area of C.S.}}{\text{area of C.S.}} = \frac{F_1}{A}$

Longitudinal strain = $\frac{\text{change in length}}{\text{original length}} = \frac{\Delta l}{l}$

$\frac{\text{Long. stress}}{\text{Long. strain}} = \text{Yung's modulus.}$
 \rightarrow depends on material.

$$Y = \frac{F_1/A}{\Delta l/l} = \frac{F(l)}{A(\Delta l)}$$

2) Shear Modulus (Rigidity modulus)



Shear stress = $\frac{F_1 \text{ (Parallel to area)}}{\text{area}}$

$$\tan \theta = \frac{x}{h} = \frac{x'}{h'}$$

$$\theta = \frac{x}{h} = \frac{x'}{h'}$$

Shear strain = $\theta = \text{angle of tilt}$
 $= \frac{x}{h} = \frac{x'}{h'}$

$\frac{\text{shear stress}}{\text{shear strain}} = \text{shear modulus } (\eta)$

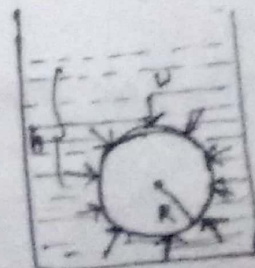
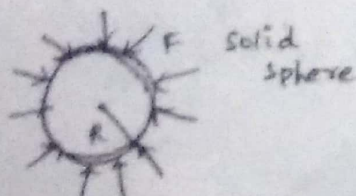
$$\eta = \frac{F_1/\text{area}}{\theta} = \frac{F_1/\text{area}}{x/h}$$

$$\eta = \frac{(F_1)(h)}{(\text{area})(x)}$$

\rightarrow For an ideal liquid,
 Bulk modulus = ∞ kg
 (vol. strain) = 0.
 Rigidity modulus = 0.
 (tangential drag) = 0.

\rightarrow For a perfect rigid body, $\eta = \infty$
 $\chi = 0, \theta = 0, \phi = 0$

3) Bulk Modulus:



Volume stress = Force acting per unit area
= Pressure.

Volume strain = $\frac{\text{change in Volume}}{\text{Original Volume}} = -\frac{\Delta V}{V}$

Bulk Modulus (B) = $\frac{\text{Volume stress}}{\text{Volume strain}}$

$$B = \frac{\Delta P}{-\frac{\Delta V}{V}} = \frac{\Delta P \cdot V}{-\Delta V}$$

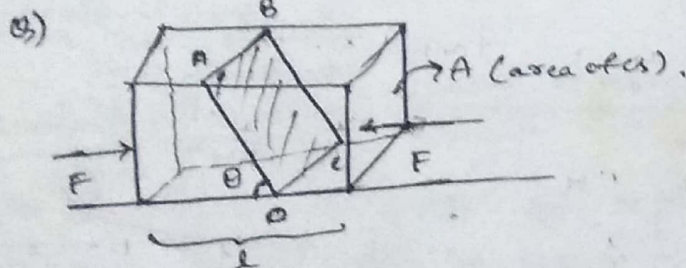
Q) If a rod has length l_1 under app. of force $3W$, and has length l_2 under app. of $4W$, Find its orig. length.

Sol: let orig. length = l

$$\frac{3(l_2 - l)}{A(l_1 - l)} = \frac{4(l_2 - l)}{A(l_1 - l)} \quad \text{[Young's modulus is const.]}$$

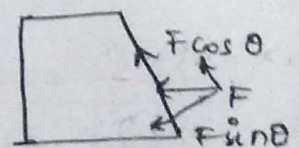
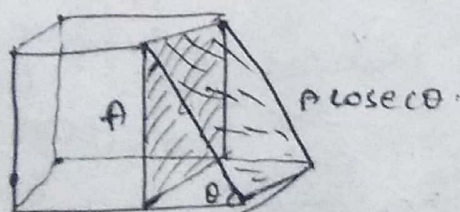
$$3l_2 - 3l = 4l_1 - 4l$$

$$l = 4l_1 - 3l_2$$



Find the longitudinal and shear stress on surface ABCD?

Sol:



$$\begin{aligned} \text{Longitudinal stress} &= \frac{F_{\perp}}{\text{area}} \\ &= \frac{F \sin \theta}{A / \sin \theta} = \frac{F \sin^2 \theta}{A} \end{aligned}$$

$$\begin{aligned} \text{Shear stress} &= \frac{F_{\parallel}}{A} \\ &= \frac{F \cos \theta}{A / \sin \theta} \\ &= \frac{F \sin 2\theta}{2A} \end{aligned}$$

If $\theta = 90^\circ$,

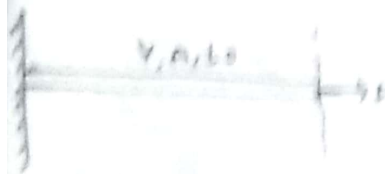
Longitudinal stress is max.

If $\theta = 45^\circ$, shear

stress is maximum.

ELASTIC POTENTIAL ENERGY STORED IN THE WIRES (STRAIN ENERGY)

(29)



For A $Y = \frac{(F)(L_0)}{(A)(x)}$

$$Y = \frac{FL_0}{Ax} \Rightarrow F = \frac{YA^2}{L_0} = \left[\left(\frac{YA}{L_0} \right) (x) = Kx \right] \left(K = \frac{YA}{L_0} \right)$$

elastic P.E stored in

the wire $= U = \frac{1}{2} Kx^2$

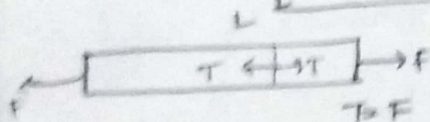
$$= \frac{1}{2} \left(\frac{YA}{L_0} \right) (x)^2 = \frac{1}{2} YA \left(\frac{x}{L_0} \right)^2 (L_0)$$

$$\Rightarrow U = \frac{1}{2} (Y) (\text{strain})^2 (\text{Volume})$$

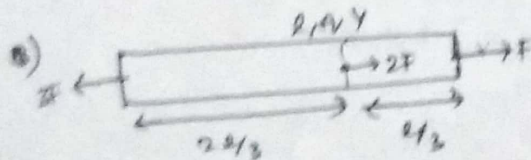
$$U = \frac{(\text{stress})(\text{strain})(\text{Volume})}{2}$$

$\frac{U}{\text{Volume}} = \text{energy density} = u$

$$u = \frac{1}{2} (\text{stress})(\text{strain})$$

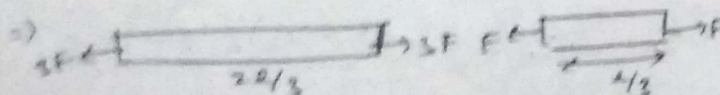
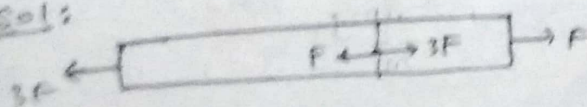


$$e = \frac{FL}{AY} \quad (e - \text{extension})$$



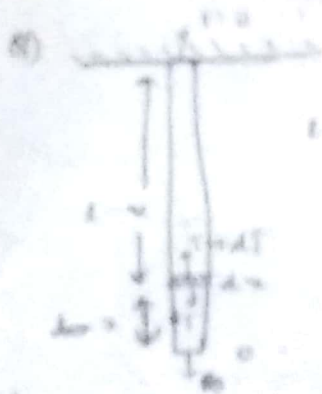
Find the extension of the rod.

Sol:



$$e = \frac{1/3 F (2L)}{2AY}$$

$$e = \frac{FL}{3AY} \Rightarrow e = \frac{7FL}{3AY}$$

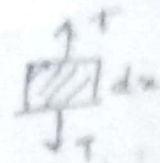


L, A, γ, W

Find the extension in the rod due to its own weight?

Sol:

$$T = \frac{(L-x)W}{L} \quad T = \left(\frac{Mx}{L}\right)g$$



$$d\theta = \frac{T \cdot dx}{A \cdot \gamma}$$

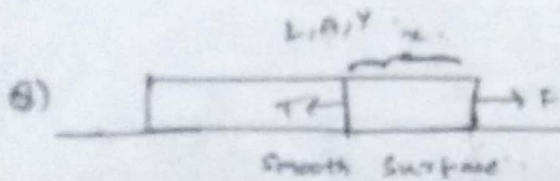
$$x = \left(\frac{Mg}{L}\right) \frac{L^2}{2}$$

$$\Delta = \int d\theta = \frac{Mg}{LA\gamma} \int x \cdot dx$$

$$= \frac{MgL}{2LA\gamma} \quad \boxed{\frac{MgL}{2A\gamma}}$$

NOTE: Since, tension is varying linearly, we can get directly by using average method

$$\frac{(0 + Mg)}{2} \cdot \frac{L}{A\gamma} = \frac{MgL}{2A\gamma}$$

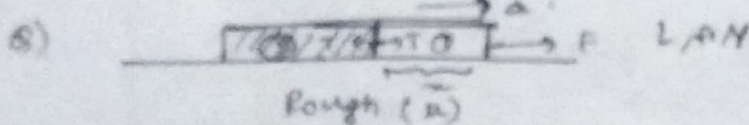


Find the extension of rod?

Sol:

$$T = F \left(1 - \frac{x}{L}\right)$$

$$\Delta = \frac{F \cdot a \cdot L}{A\gamma} = \frac{FL}{2A\gamma}$$

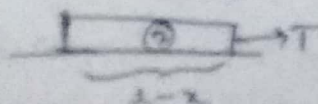


Find the extension in the Rod.

Sol:

$$F_{\text{net}} = \left(\frac{W}{L} - \mu \frac{W}{L}\right) \cdot L$$

$$a = \frac{F - \mu mg}{m}$$



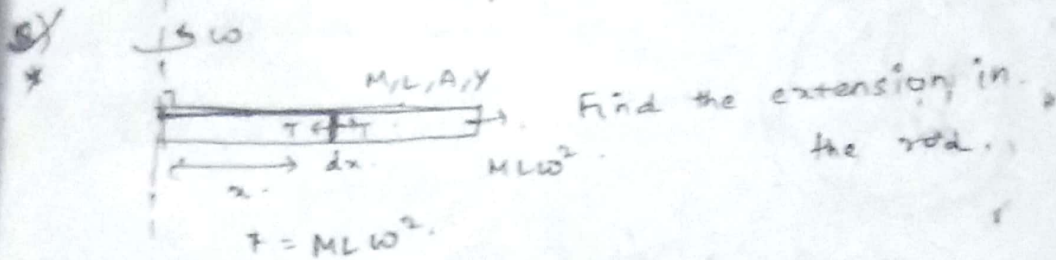
$$T = \left(\frac{L-x}{L}\right)ma - \mu \left(\frac{L-x}{L}\right)mg$$

$$x = \left(\frac{F - \mu mg}{L}\right) \cdot L$$

$$T = \left(\frac{L-x}{L} \right) m \left(\frac{1}{2}g + F/m - \frac{1}{2}g \right)$$

$$\Rightarrow \frac{F(L-x)}{L}$$

$$e = \frac{\left(0 + \frac{F}{2} \right) L}{AY} = \frac{FL}{2AY}$$



$$e = \frac{\left(\frac{0+F}{2} \right) L}{AY} = \frac{FL}{2AY} = \frac{ML\omega^2(L)}{2AY}$$

$= M$

$$T = \left(\frac{Mx}{L} \right) \cdot x \cdot \omega^2$$

$$de = \frac{Mx^2\omega^2}{L} \cdot dx$$

$$\Rightarrow \int de = e = \frac{M\omega^2}{L} \int_0^L x^2 dx$$

$$\Rightarrow \frac{M\omega^2 L^3}{3L} = \frac{M\omega^2 L^2}{3}$$

Method;

$$dT = (-dm)(x)(\omega^2)$$

$$= -\left(\frac{M}{L} dx \right) (x)(\omega^2)$$

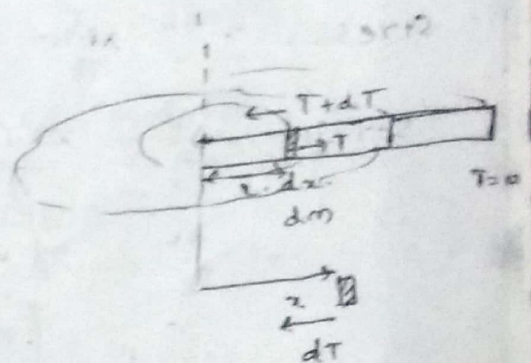
$$\int_0^L dT = -\frac{M\omega^2}{L} \int_0^L x dx$$

$$T = -\frac{M\omega^2}{L} \left[\frac{x^2}{2} \right]_0^L$$

$$T = \frac{M\omega^2}{2L} (L^2 - x^2)$$

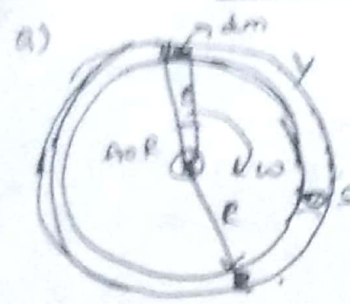
$$de = \frac{T \cdot dx}{AY} = \frac{M\omega^2}{2L} (L^2 - x^2) \cdot dx$$

$$e = \frac{M\omega^2}{2L} \int_0^L (L^2 - x^2) dx$$



$$I = \frac{M\omega^2}{2\lambda Y} \left(L^3 - \frac{L^3}{3} \right) \Rightarrow \frac{M\omega^2}{2\lambda Y} \left(\frac{2L^3}{3} \right)$$

$$e = \frac{M\omega^2 L^2}{3\lambda Y}$$



If Ring rotates with angular velocity ' ω ' about an AOR passing through centre and \perp to ring,
 λ - mass per unit length
 Y - Young's modulus,
 S - area of CS

What is the increase in radius of the ring?

Sol.

$$2T \sin(\theta/2) = dm R \omega^2$$

$$2T \sin(\theta/2) = \lambda (R\theta) \cdot R \omega^2$$

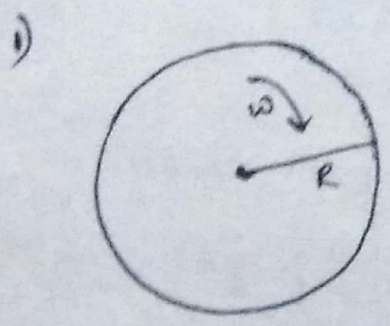
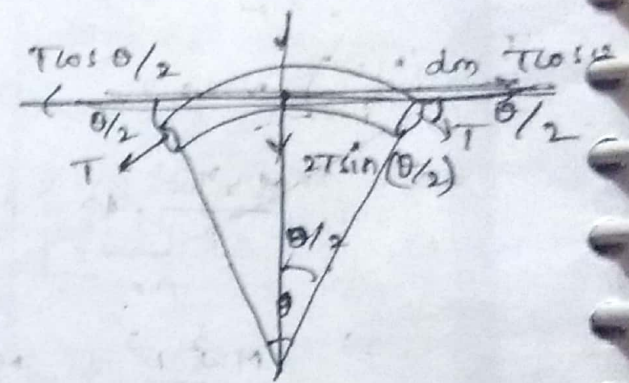
$$T = \lambda R^2 \omega^2$$

$$\text{Stress} = \frac{T}{S} = \frac{\lambda R^2 \omega^2}{S}$$

$$\text{Strain} = \frac{\Delta R}{R} = \frac{2\pi(R + \Delta R) - 2\pi R}{2\pi R} = \frac{\Delta R}{R}$$

$$\frac{\Delta R}{R} = \frac{\lambda R^2 \omega^2}{SY}$$

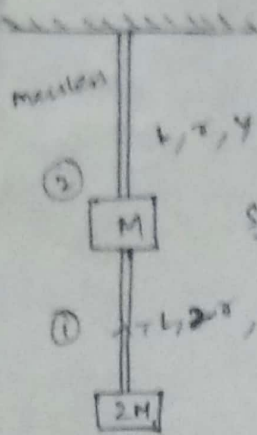
$$\Delta R = \frac{\lambda R^3 \omega^2}{SY}$$



If Breaking stress of material is X ,
 Find the max. angular velocity of the ring allowed not to rupture it.

Sol.

$$X \geq \frac{\lambda R^2 \omega^2}{S} \Rightarrow \frac{XS}{\lambda R^2} \geq \omega^2 \Rightarrow \omega \leq \sqrt{\frac{XS}{\lambda R^2}}$$



Find the total elastic P.E stored in the system.

Sol. (1) $T = 2Mg$

$$\text{Stress} = \frac{2Mg}{\pi(Ar^2)}$$

$$\text{Strain} = \frac{2Mg}{\pi(Ar^2)(Y)}$$

$$\text{Volume} = \pi(L)(Ar^2)$$

$$(EPE)_1 = \frac{1}{2} \times \frac{M^2 g^2}{\pi^2 \times 16r^4} \times \pi(L)(Ar^2)$$

$$= \frac{M^2 g^2 (L)}{2\pi r^2 Y}$$

(2) $(EPE)_2$
 $T =$

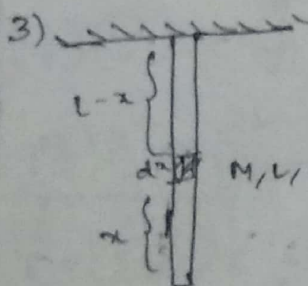
Sol: $U = \frac{1}{2} (\text{Stress}) (\text{Strain}) (\text{Vol.})$

$$= \frac{1}{2} \times \frac{F^2 L}{AY}$$

$$F_2 = 3mg \Rightarrow U_2 = \frac{9m^2 g^2 L}{2 \times \pi r^2 \times Y} = \frac{9m^2 g^2 L}{2\pi r^2 Y}$$

$$F_1 = 2mg \Rightarrow U_1 = \frac{4m^2 g^2 L}{2 \times \pi \times 4r^2 Y} = \frac{m^2 g^2 L}{\pi r^2 Y}$$

$$U_{\text{tot}} = U_1 + U_2 = \frac{5m^2 g^2 L}{\pi r^2 Y}$$



Find E.P.E stored in the wire.

$$e = \frac{Mg L}{2AY}$$

Sol: $U = \frac{1}{2} (Y) (\text{Strain})^2 (\text{Vol.})$

$$= \frac{1}{2} \times Y \times \frac{M^2 g^2}{A^2 Y} \times A \times L = \frac{M^2 g^2 L}{2AY}$$

~~$$\frac{M^2 g^2 L}{2AY}$$~~

$$T = \left(\frac{Mg}{L} \right) (x) \quad \text{stress} = \left(\frac{Mg}{AL} \right) (x)$$

Energy stored in elemental part -

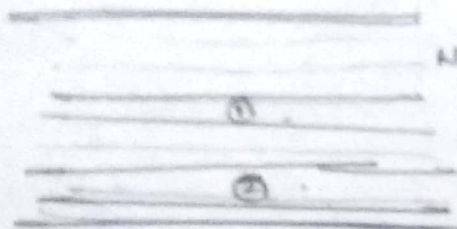
$$dU = \frac{1}{2} \times \frac{(\text{stress})^2}{Y} \times \text{Vol.}$$

$$= \frac{1}{2} \times \left(\frac{M^2 g^2}{A^2 L^2} \times x^2 \right) (A dx)$$

$$= \frac{M^2 g^2}{2 A L^2 Y} x^2 dx$$

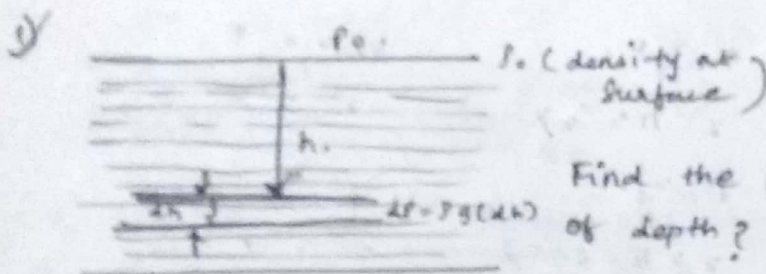
$$\Rightarrow \int_0^L dU = \int_0^L x^2 dx \cdot \left(\frac{M^2 g^2}{2 A L^2 Y} \right)$$

$$U = \frac{M^2 g^2 \times L^3}{6 A L^2 Y} = \frac{M^2 g^2 L}{6 A Y}$$



Non ideal fluid
(compressible)

$$P_2 > P_1$$



Find the pressure as function
of depth? Bulk Modulus of liquid - B.

Sol: $B = \frac{\Delta P}{-\frac{\Delta V}{V}}$

$$B = \frac{dP}{\frac{dV}{V}} \times V$$

$$\boxed{B \left(\frac{dP}{P} \right) = dP}$$

$$B \left(\frac{dP}{P} \right) = \rho \cdot g \cdot dh$$

$$B \int_{P_0}^P \frac{dP}{P} = g \int_0^h dh$$

$$B \cdot \left[\ln P \right]_{P_0}^P = gh \Rightarrow$$

$$\boxed{B \left(-\frac{1}{P} + \frac{1}{P_0} \right) = gh} \quad \text{--- (2)}$$

$$\frac{dP}{dV} = -\frac{m}{V^2} \quad P = m/V$$

$$\frac{dP}{dV} = \left(-\frac{m}{V} \right) \left(\frac{1}{V} \right)$$

$$\frac{dP}{dV} = -P \left(\frac{1}{V} \right)$$

$$\boxed{\left(-\frac{dV}{V} \right) = \frac{dP}{P}} \quad \text{--- (1)}$$

$$\int_{P_0}^P \frac{dP}{P} = \int_{l_0}^l \frac{dl}{l}$$

$$P - P_0 = B(\log P/P_0)$$

$$P = P_0 + B \ln(P/P_0) \quad - (3)$$

$$\Rightarrow P = P_0 + B \ln \left(\frac{B}{B - g h P_0} \right)$$

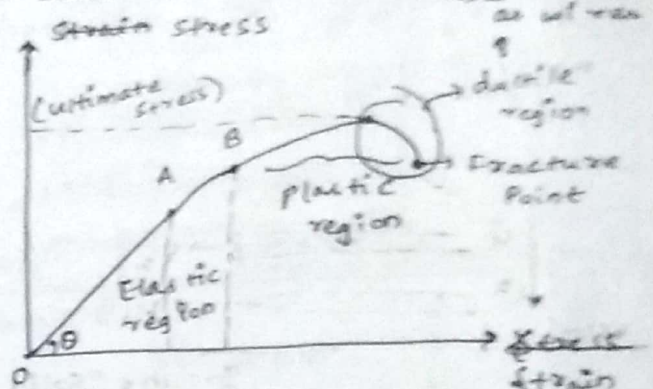
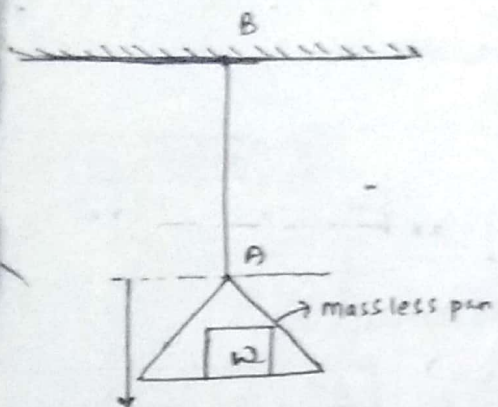
From Eq - (2)

$$\frac{1}{P} = \frac{1}{P_0} - \frac{g h}{B}$$

$$l = \frac{B P_0}{B - g h P_0}$$

$$\Rightarrow \frac{P}{P_0} = \frac{B}{B - g h P_0}$$

Behaviour of metal wire under increasing Load:



→ A material is said to be more brittle if its Fracture point is more closer to Ultimate stress point.

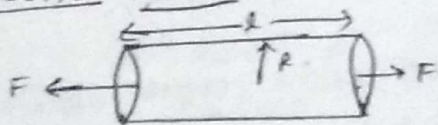
Stress & strain (Hooke's Law)

A - Proportionality limit.

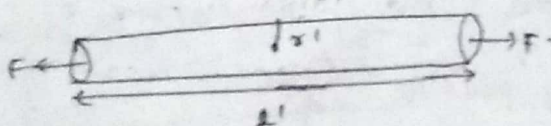
B - Elastic limit.

$$\text{Slope} = \tan \theta = \frac{\text{stress}}{\text{strain}} = Y$$

Poisson's Ratio:



$$\frac{\Delta l}{l} = \text{longitudinal strain}$$



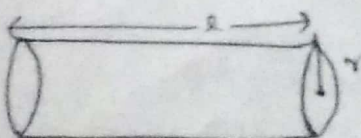
$$\frac{\Delta r}{r} = \text{lateral strain (Transverse strain)}$$

$$l' > l, r' < r$$

$$\text{Poisson's Ratio} = (\sigma) = \frac{\text{Lateral strain}}{\text{longitudinal strain}}$$

$$\sigma = \frac{-\Delta r / r}{\Delta l / l}$$

$$\frac{1}{\sigma} = \frac{\Delta l / l}{-\Delta r / r} \Rightarrow \left[\frac{-\Delta r}{r} = \sigma \left(\frac{\Delta l}{l} \right) \right]$$



$$\text{Volume (V)} = \pi r^2 l$$

$$V = \pi r^2 l$$

$$\frac{1}{v} dv = 0 + 2 \cdot \frac{dr}{r} + \frac{dl}{l}$$

$$\frac{dv}{v} = 2 \cdot \left(-\sigma \cdot \frac{dl}{l} \right) + \frac{dl}{l}$$

$$\boxed{\frac{dv}{v} = (1 - 2\sigma) \frac{dl}{l}}$$