

On Simple Planar Variable Geometry Trusses

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1 Introduction

A *variable geometry truss* is a truss in which changes shape by means of the change of the length of members, in contrast to many robot arms which change joint angles. We call a member which can change length an *actuator*. A truss constructed purely by starting with a triangle and repeatedly adding two members and a new joint to one side of an existing triangle to form a new triangle is a *simple truss*. This paper is concerned only with simple planar trusses. In particular, we focus on trusses isomorphic to the Warren truss (e.g., an unbranching configuration of triangles.) Furthermore, although the word *truss* connotes forces and structural analysis, we are concerned purely with geometry. Although motivated by robotics, we assume that our trusses and actuators are strong enough that we need not consider the forces acting upon the truss—in other words, we are treating it as mechanism and not a machine.

If we imagine a joint, usually at one end of our truss, to be an end effector, the fundamental goal of this paper is to answer the question:

How does a change in length of an actuator change the position of an end effector?

2 Formulation

A *truss* structure is a graph and a set of fixed nodes $T = (G, F)$. $G = (V, E)$. V is a set of nodes or joints. E is a set of lines or members which are 2-element subsets of V . At least two nodes are considered to be fixed in space via a set of fixed nodes $F = \{(i, x, y) \mid i \in V, x, y \in \mathbb{R}\}$.

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We number joints from zero and designate them with a subscript. Members are designated by two subscripts, naming the joints they connect.

A *configuration* is a function mapping each member of a truss structure to a non-negative length. $C : V \times V \mapsto \mathbb{R}$.

A *geometry* is a placement of each joint in the Cartesian plane. $G : V \mapsto \mathbb{R} \times \mathbb{R}$.

A *simple truss* is a truss constructed from a triangle by adding a single joint and two members to a side repetitively. A *Warren truss* is an unbranched simple truss, which is isomorphic to a truss that is a single chain of equilateral triangles. Furthermore, we insist that each even node n occurs as a anti-clockwise turn (in the anti-clockwise semiplane) from the vector $\overrightarrow{n-2, n-1}$, and each odd node occurs as a clockwise turn (in the clockwise semiplane) from the previous to nodes.

For a Warren truss, there is a simple algorithm W for computing a geometry from a configuration: $W : C \mapsto G$.

A goal joint is a joint j such that $j \in V$ and a desired position given by a goal function $d : V \mapsto \mathbb{R}^2$. In robotics, a goal joint is often called an *end effector*.

A *scoring function* is a function that takes a geometry and returns a real, non-negative value based on the goal node positions given by a goal function and the actual position given by a Geometry. A score of zero is considered a perfect score and the a higher score represents a less sought-after result.

A *linear distance scoring function* is a special scoring function that takes a geometry and returns a real, non-negative value based solely on a summation of distances between the goal node positions given by a goal function and the actual position given by a Geometry.

A truss *problem* $P = (T, d, s)$ is a truss with a scoring function and a desired position function. A solution to a problem is a configuration and a geometry. An optimal solution is a solution which minimizes the score.

Our fundamental goal is to develop a formula for the partial derivative of the end effector with respect to the change in length of a member.

A further goal is to be able to render a diagram illustrating the impact on the end-effector of a change of each member, as drawn by rendering a vector from the center point of each member.

A further goal is to have algorithms to determine:

- What is the minimum overall change in length to a group of actuators to solve a Problem?
- Can a Problem be solved with a single change in length?
- Assuming bounds on the lengths of actuators, can we solve Problems?
- What is the workspace of a truss?

In the remainder of this paper we will consider only Warren trusses, linear distance scoring functions, and desired position functions that map only one node, called an end effector, to a desired position. Furthermore, we will assume that the first two nodes are fixed and that the furthest node (by path length) from the first node is the end effector.

3 Moving an External Member

We seek a formula for the change in position of the end effector e with respect to change in the length of a member given a geometry. In the case of a Warren truss, all members can be divided between external members and internal members. A change to the length of an external member is particularly simple.

The change in position is centered on the goal node representing the place the goal node would move to with a unit change in length. However, the derivative is only valid as an infinitesimal, but as an infinitesimal its magnitude and direction may be usefully added to or compared to other such vectors. We can thus usefully tell which member would change the position of the goal node most rapidly in response to a minute change in different members.

The fundamental observation is that for an external member (C, A) , a change in length generates a rotation θ about joint B whose position is given by (B_x, B_y) . θ is the angle of the vector from the pivot point B to the end-effector E with the x -axis. This rotation applies to the triangle defined by three joints: $\triangle B, C, E$. Note that this triangle does not exist as a physical structure in the truss.

By using the law of cosines, where $a = \|\vec{A, B}\|$, $b = \|\vec{A, C}\|$, $c = \|\vec{B, C}\|$, where b is the member opposite the pivot joint B which changes the angle $\phi_{i-1} = \angle ABC$. In other words, ϕ is a signed angle measure BA moved into BC , with positive representing anti-clockwise.

$$\cos \phi = \frac{a^2 - b^2 + c^2}{2ac}$$

or

$$\phi = \arccos \frac{a^2 - b^2 + c^2}{2ac}$$

Using Wolfram Alpha to differentiate this, we obtain:

$$\frac{\partial \phi}{\partial b} = \frac{b}{ac \sqrt{1 - \frac{(a^2 + c^2 - b^2)^2}{4a^2c^2}}}$$

This derivation loses the sign information, which we recover by considering whether $S = \text{sign } \phi$.

The change in the end effector is always perpendicular to the drawn from the pivot joint to the effector and proportional to its length. An alternative way of looking at this

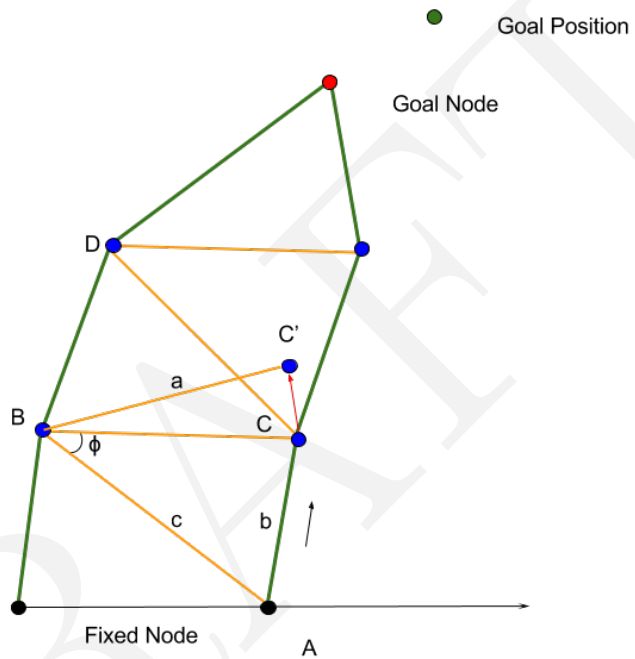


Figure 1: A Truss Problem with Change to an External Member Length

is that the result should be orthogonal to the vector

$$\begin{bmatrix} (e_x - x) \\ (e_y - y) \end{bmatrix}$$

In other words:

$$S \cdot \begin{bmatrix} -(e_y - y) \\ (e_x - x) \end{bmatrix}$$

With the sign S is negative if ϕ is clockwise. ϕ cannot be zero or equal or exceed π in a physical machine is excluded for that reason.

Since the change in ϕ is equal to the change in θ , we can use the chain rule:

$$\frac{\partial e}{\partial b} = \begin{bmatrix} \frac{\partial e_x}{\partial b} \\ \frac{\partial e_y}{\partial b} \end{bmatrix} = \frac{Sb}{ac\sqrt{1 - \frac{(a^2 + c^2 - b^2)^2}{4a^2c^2}}} \begin{bmatrix} -(e_y - y) \\ (e_x - x) \end{bmatrix}$$

So this is a closed-form expression for the change in the position of an end effector for any external member $i, i - 2$ we choose. If we then know how the scoring functions changes as the end effector moves, we can compute the change in the scoring functions as we change $b = \|AC\|$, which is what we need for numeric optimization.

4 Moving an Internal Member

In the Internal Member Length change diagram, $\angle ABC = \beta$, $\angle CBD = \psi$, $\angle BCD = \chi$, $\angle ACB = \gamma$, $\angle BDC = \delta$, and $\angle BAC = \alpha$. The lengths are marked a, b, c, f, g , with a being opposite node A and a diagonal of the quadrilateral $ABDC$.

Furthermore, $\angle ABD = \beta + \psi$, and $\angle ACD = \gamma + \chi$.

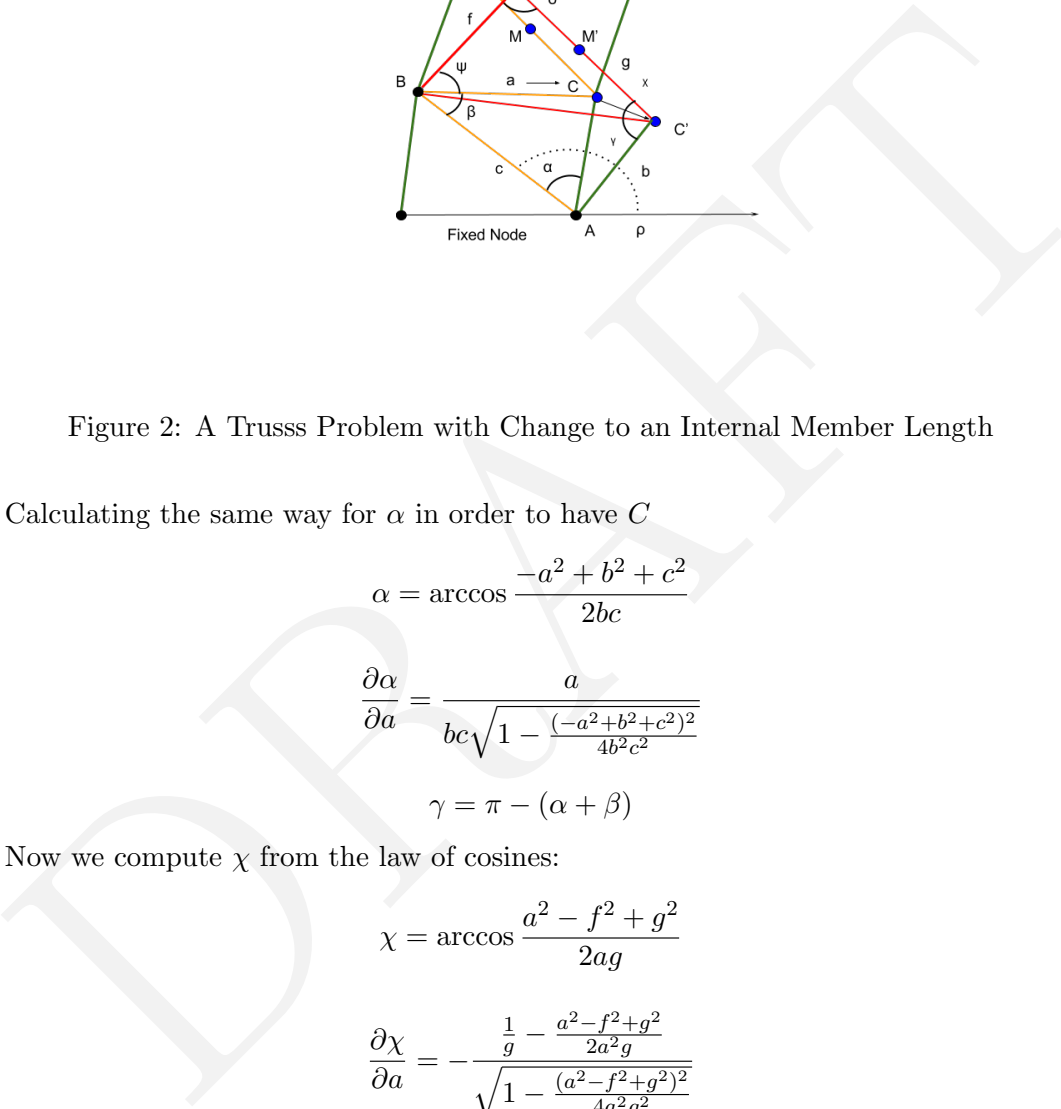
The absolute angle between the line between the fixed nodes A and B and the x -axis is ρ (counting positive as anticlockwise from the x -axis.) Both A and B are considered fixed, but both C and D move as a changes. D rotates about B and C rotate about A .

Because the member DC does not change its length, we conceptualize the impact of a change to length a as translating between C and D and rotating BC about M . By knowing the change in the absolute rotation and the change in the position of M as a changes infinitessimally, we know how the end effector E changes.

We can compute $\frac{\partial \beta}{\partial a}$ directly (using Wolfram Alpha) from ϕ :

$$\beta = \arccos \frac{a^2 - b^2 + c^2}{2ac}$$

$$\frac{\partial \beta}{\partial a} = -\frac{\frac{1}{c} - \frac{a^2 - b^2 + c^2}{2a^2c}}{\sqrt{1 - \frac{(a^2 - b^2 + c^2)^2}{4a^2c^2}}}$$



Calculating the same way for α in order to have C

$$\alpha = \arccos \frac{-a^2 + b^2 + c^2}{2bc}$$

$$\gamma = \pi - (\alpha + \beta)$$

$$\chi = \arccos \frac{a^2 - f^2 + g^2}{2ag}$$

$$\begin{bmatrix} C_x \\ C_y \end{bmatrix} = \begin{bmatrix} A_x + b \cos (\rho - \alpha) \\ A_y + b \sin (\rho - \alpha) \end{bmatrix}$$

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$$\begin{bmatrix} \frac{\partial C_x}{\partial a} \\ \frac{\partial C_y}{\partial a} \end{bmatrix} = \begin{bmatrix} 0 + \frac{\partial b \cos(\rho - \alpha)}{\partial a} \\ 0 + \frac{\partial b \sin(\rho - \alpha)}{\partial a} \end{bmatrix} = \begin{bmatrix} b \frac{a \sin(\rho - \alpha)}{bc \sqrt{1 - \frac{(-a^2 + b^2 + c^2)^2}{4b^2 c^2}}} \\ -b \frac{a \cos(\rho - \alpha)}{bc \sqrt{1 - \frac{(-a^2 + b^2 + c^2)^2}{4b^2 c^2}}} \end{bmatrix} = \frac{a}{c \sqrt{1 - \frac{(-a^2 + b^2 + c^2)^2}{4b^2 c^2}}} \begin{bmatrix} \sin(\rho - \alpha) \\ -\cos(\rho - \alpha) \end{bmatrix}$$

The angle θ between CD and the x -axis is therefore given by:

$$\begin{aligned} \theta &= \\ &(\rho - \alpha) + (\pi - (\chi + \gamma)) = \\ \rho + \pi - (\alpha + \chi + (\pi - (\alpha + \beta))) &= \\ \rho + \pi - (\chi + (\pi - \beta)) &= \\ \rho + -\chi + \beta \end{aligned}$$

We seek $\frac{\partial \theta}{\partial a}$. However, χ and β , computed from arccos, are “internal angles”. We need to seek absolute orientation.

In computing all of this, one must be very careful to keep the signs straight.

In fact, the cleanest way to keep angles straight is to convert angles quickly from “internal angles” between two vectors and compute the “absolute orientation” or vectors (that is, the angle measure anticlockwise against the x -axis).

The computation of $\frac{\partial \theta}{\partial a}$ above should be considered the computation of an “absolute orientation”.

We can think of the computation of $\theta = \rho - \chi + \beta$ is a zig zag computation. To find the change in the rotation of θ , we find the change in χ and then the change in β .

We will say the “sense” of an angle $\angle XYZ$ is clockwise or anticlockwise depending on if the smallest turn from X to Z through Y is clockwise or anticlockwise.

$$\text{sense } \angle XYZ = \begin{cases} 0 & \angle XYZ = \pi \\ 1 & \text{if } \angle XYZ \text{ is anticlockwise moving } \overrightarrow{Y, X} \text{ into } \overrightarrow{Y, Z} \\ -1 & \text{otherwise} \end{cases}$$

The orientation $\tau = \overrightarrow{B, C}$ is thus $\rho + -(\pi - \|\beta\|)$ if $\angle ABC$ is clockwise, and $\rho + (\pi - \|\beta\|)$ if $\angle ABC$ is anticlockwise.

The orientation $\theta = \overrightarrow{C, D}$ is thus $\tau + -(\pi - \|\chi\|)$ if $\angle BCD$ is clockwise, and $\tau + (\pi - \|\chi\|)$ if $\angle BCD$ is anticlockwise.

$$\begin{aligned} \theta &= \rho + (\text{sense } \angle ABC)(\pi - \|\beta\|) + (\text{sense } \angle BCD)(\pi - \|\chi\|) \\ &= \rho + (\text{sense } \angle ABC)\pi + (\text{sense } \angle BCD)\pi + -(\text{sense } \angle ABC)\|\beta\| + -(\text{sense } \angle BCD)\|\chi\| \end{aligned}$$

The quantities π and $\text{sense } \angle ABC$ and $\text{sense } \angle BCD$ don't depend on a .

$$\begin{aligned}\frac{\partial \theta}{\partial a} &= 0 + -(\text{sense } \angle ABC) \frac{\partial \beta}{\partial a} + -(\text{sense } \angle BCD) \frac{\partial \chi}{\partial a} \\ &= -(\text{sense } \angle ABC) \frac{\frac{1}{c} - \frac{a^2 - b^2 + c^2}{2a^2 c}}{\sqrt{1 - \frac{(a^2 - b^2 + c^2)^2}{4a^2 c^2}}} + -(\text{sense } \angle BCD) \frac{\frac{1}{g} - \frac{a^2 - f^2 + g^2}{2a^2 g}}{\sqrt{1 - \frac{(a^2 - f^2 + g^2)^2}{4a^2 g^2}}}\end{aligned}$$

Now E is simply a translation by $\frac{\partial C}{\partial a}$ by a rotation about C , which as we have previously shown

$$\begin{aligned}\frac{\partial e}{\partial a} &= \begin{bmatrix} \frac{\partial C_x}{\partial a} \\ \frac{\partial C_y}{\partial a} \end{bmatrix} + \frac{\partial \theta}{\partial a} \begin{bmatrix} -(e_y - C_y) \\ (e_x - C_x) \end{bmatrix} \\ \frac{\partial e}{\partial a} &= \frac{a}{c \sqrt{1 - \frac{(a^2 + b^2 + c^2)^2}{4b^2 c^2}}} \begin{bmatrix} \sin(\rho - \alpha) \\ -\cos(\rho - \alpha) \end{bmatrix} + \frac{\partial \theta}{\partial a} \begin{bmatrix} -(e_y - C_y) \\ (e_x - C_x) \end{bmatrix}\end{aligned}$$

5 Testing

The best way to test this is to build a diagram that draws the vector $\frac{\partial e}{\partial l_{BC}}$ at the center of each edge \vec{BC} . This should allow both the direction and magnitude to be visualized in a reasonable way. We could call this a *delta diagram*.

6 References