

### Example 14-8 (From Cengel, Chapter 14)

Given

$$\beta_{2_s} := 60 \text{ deg} \quad c_x := 47.1 \frac{\text{m}}{\text{s}} \quad \omega := 1750 \text{ rpm}$$

Required

$$\beta_{2_r} = ? \quad \beta_3 = ?$$

Assumptions:

- Flow is steady in the middle
- Incompressible flow
- Constant cross sectional area ( $c_x = \text{const.}$ )
- No whirl at exit ( $\alpha_3 = 0$ )

$$\tan(\beta_{2_r}) = \frac{W_{2t}}{c_x} = \frac{c_{2\theta} - U}{c_x} = \frac{c_{2\theta} - \omega \cdot r}{c_x}$$

$$\text{And... } \tan(\beta_{2_r}) = \frac{c_{2\theta}}{c_x} \xrightarrow{\text{solve, } c_{2\theta}} \frac{\text{float, 3}}{c_x} = \frac{47.1 \cdot \text{m} \cdot \tan(60.0 \cdot \text{deg})}{\text{s}} = 81.58 \frac{\text{m}}{\text{s}}$$

$$c_{2\theta} := 81.58 \frac{\text{m}}{\text{s}}$$

$$\tan(\beta_{2_r}) = \frac{c_{2\theta} - \omega \cdot r}{c_x}$$

$$\beta_{2_r} := \text{atan}\left(\frac{c_{2\theta} - \omega \cdot r}{c_x}\right) = 9.966 \text{ deg}$$

If we design the blades so that there's no whirl at the exit, then  $c_3 = c_x$  meaning...

$$\tan(\beta_3) = \frac{\omega \cdot r}{c_x} \xrightarrow{\text{solve, } \beta_3} \frac{\text{float, 6}}{c_x} \rightarrow \text{atan}(14.862 \cdot \text{rpm} \cdot \text{s}) = 57.278 \text{ deg} \quad \beta_3 := 57.278 \text{ deg}$$

Cengel's Answer:

We conclude that the rotor blade at this radius has a leading edge angle of about **73.1°** (Eq. 3) and a trailing edge angle of about **57.3°** (Eq. 4). A sketch of the rotor blade at this radius is provided in Fig. 14-65; the total curvature is small, being less than 16° from leading to trailing edge.

