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DME LAB-4

$$FOS = 2$$

$$T = \frac{60 \text{ (Power)}}{2\pi N} = \frac{60 \times 50,000}{2\pi \times 1000} = 477.464829$$

$$\text{Torque} \approx 477.5$$

$$\text{Torque}_{\text{design}} = 477.5 \times 2 = 955 \text{ N.m}$$

(S.F)

Shaft Diameter (shear stress)

$$\tau = \frac{T}{J} R$$

$$\tau = \frac{T}{\frac{\pi}{32} (d_h^4 - d^4)}$$

$$\tau = \frac{955 \times d}{\frac{\pi}{32} (16d^4 - d^4)}$$

$$\text{put } d_h = 2d$$

$$\sigma_{yt} = 400 \text{ N/mm}^2$$

$$\tau = 0.5 \sigma_{yt}$$

$$\tau = \frac{955 \times d}{\frac{\pi}{32} \cdot 15 d^4}$$

$$\tau = \frac{955 \times 32}{15 \times \pi \times d^3}$$

$$d = \left(\frac{955 \times 32}{15 \times \pi \times \frac{400}{2}} \right)^{\frac{1}{3}}$$

$$d = 1.86482209 \text{ mm}$$

$$d \approx 1.865 \text{ mm}$$

shaft =

→

$$\frac{200}{2} = \frac{955 \times \frac{d}{2} \times 1000}{\frac{\pi}{32} d^4}$$

$$d^3 = \frac{955 \times 32}{2 \times \pi \times 1000} \times 1000$$

$$d \approx 3.65 \text{ mm}$$

$$d = 36.5 \text{ mm}$$

$$d_h = 7.3 \text{ mm}$$

$$l_h = 54.75 \text{ mm}$$

$$D = 109.5 \text{ mm}$$

$$t = 18.25 \text{ mm}$$

$$t_1 = 9.125 \text{ mm}$$

$$d_r = 54.75 \text{ mm}$$

$$D_o = 164.25 \text{ mm}$$

$$\text{no of bolts} = 3$$

Key

$$b = 9.125 \text{ mm}$$

$$A =$$

$$d_h = 73 \text{ mm}$$

$$l_h = 54.75 \text{ mm}$$

$$D = 109.5 \text{ mm}$$

$$t = 18.25 \text{ mm}$$

$$t_1 = 9.125 \text{ mm}$$

$$d_r = 54.75 \text{ mm}$$

$$D_o = 164.25 \text{ mm}$$

$$\text{no of bolts} = 4$$

Key

$$b = 9.125 \text{ mm}$$

$$h = 6.083 \text{ mm}$$

$$L = 54.75 \text{ mm}$$

Standard

$$\text{Key} = (12 \times 8) - 5 \text{ depth.}$$

2) Design of flange

$$t_f = d/2 = 36.5/2 = 18 \text{ mm}$$

$$T = \pi D \times t_f \times \tau_c \times D/2$$

$$10^3 \times 477.5 = \pi \times 3d \times \frac{d}{2} \times \tau_c \times \frac{3d}{2}$$

$$10^3 \times 477.5 = \pi \times \frac{9}{4} \times d^3 \times \tau_c$$

$$\tau_c = \frac{477.5 \times 4 \times 10^3}{9 \times \pi \times (36.5)^3} = \frac{1.389 \text{ MPa}}{1.389 \text{ N/mm}^2}$$

$$\tau_c > \tau_{\text{allowable}}$$

3) Design of bolt

$$\tau = \frac{2T}{bld} = \frac{955 \times 1000}{9 \times 54 \times 36.5} = 53.83 \text{ N/mm}^2$$

Overstressing Area

$$\sigma_c = \frac{hT}{dh d} = \frac{2 \times 955 \times 1000}{36.5 \times 54 \times 6} = 161.5 \text{ N/mm}^2$$