

## AN EQUIVALENT DOMAIN INTEGRAL METHOD FOR THREE-DIMENSIONAL MIXED-MODE FRACTURE PROBLEMS

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**Abstract**—A general formulation of the equivalent domain integral (EDI) method for mixed-mode fracture problems in cracked solids is presented. The method is discussed in the context of a 3-D finite-element analysis. The  $J$ -integral consists of two parts: the volume integral of the crack front potential over a torus enclosing the crack front and the crack surface integral due to the crack front potential plus the crack-face loading. In mixed-mode crack problems the total  $J$ -integral is split into  $J_I$ ,  $J_{II}$ , and  $J_{III}$ , representing the severity of the crack front in three modes of deformation. The direct and decomposition methods are used to separate the modes. These two methods were applied to several mixed-mode fracture problems in isotropic materials. Several pure and mixed-mode fracture problems were analyzed and results found to agree well with those available in the literature. The method lends itself to be used as a post-processing subroutine in a general purpose finite-element program.

### INTRODUCTION

SEVERAL numerical techniques, in conjunction with finite-element (FE) analyses, have been developed to calculate fracture mechanics parameters (stress-intensity factor  $K$ , strain energy release rate  $G$ , and  $J$ -integral). Three of these techniques are: (1) the virtual crack extension (VCE) method [1–4], (2) the virtual crack closure technique (VCCT) [5–8], and (3) the  $J$ -integral method [9–12]. The VCCT method is simple and accurate but can be applied only to linear elastic problems. In contrast, the VCE and  $J$ -integral methods can be applied to both linear and nonlinear problems. These methods are best demonstrated for pure mode problems or for calculating the total crack driving forces ( $G$  or  $J$ ). Application of these methods to mixed-mode fracture problems is complex. The VCE method involves a physical extension of the crack front by a small amount. Selection of the amount of crack extension is arbitrary and can introduce errors in inelastic problems. Further, both VCCT and VCE methods require an FE mesh that is nearly normal to the crack front. Except for some simple cases, generating such customized FE models for irregular-shaped cracks is difficult and time consuming, if not impossible. Furthermore, such detailed modeling may not improve the global accuracy of the boundary value solution. Therefore, pursuit of methods that do not have these limitations continues.

The  $J$ -integral method is very attractive, particularly for nonlinear material problems. With the original  $J$ -integral equation by Rice [9], Cherepanov [10, 12], and Eshelby [11] for two-dimensional (2-D) problems as the starting point, several crack tip integrals were developed to include body forces due to thermal and magnetic fields and unloading effects in elastic-plastic problems [13–18]. For 2-D problems, the crack tip integrals are written as the sum of a remote line integral and an area integral around the crack tip. For 3-D problems, the  $J$ -integral is the sum of a remote surface integral and a volume integral around the crack front. These integral formulations suffer from a common drawback in that they require the evaluation of surface integrals which include singular terms. The evaluation of these surface integrals, although possible, is very unwieldy in FE analyses.

The  $J$ -integral formulation has been modified into a domain integral form [19–23] after de Lorenzi [24, 25] introduced an S-function concept to define the virtual crack extension in 3-D cracked solids. In this method, the surface integrals for 3-D problems can be transformed into integrals over a domain or volume and, hence, the name equivalent domain integral (EDI). The EDI formulation is computationally very appealing and efficient.

Recently, Nikishkov and Atluri [20] presented an EDI formulation for cracked 3-D solids. To stimulate the singularities at the crack front, they used quarter-point singularity elements. While they present the EDI formulation in a comprehensive manner, some details of the formulation need additional explanation. Also the formulation of ref. [20] may not be general enough for problems where crack faces are subjected to external loading. The first objective of this paper is to present a general formulation of the EDI method for the calculation of the  $J$ -integral under mixed-mode loading conditions.

Most investigators use collapsed quarter-point singularity elements at the crack front to simulate the crack front singularity with a polar arrangement of elements around the crack front. This type of mesh may be suitable but not convenient, particularly for crack extension studies. Furthermore, in the plastic range the quarter-point singular element produces a  $1/r$  type singular strain field, which is valid only for elastic-perfectly-plastic material. Therefore, the second objective of this paper is to study the accuracy of the results when non-singular elements with a rectilinear arrangement of elements are used at the crack front.

First the EDI formulation for general mixed-mode fracture problems in elastic, elastic-plastic, and anisotropic materials is presented. Next, the validity of the formulation is studied by applying it to several linear elastic and isotropic mixed-mode fracture problems. Numerical implementation of the EDI method for 20-noded and eight-noded, 3-D isoparametric elements is presented in the Appendix. Several differences between the present formulation and those in the literature are highlighted.

CRACK FRONT AND DOMAIN INTEGRALS

The  $J$ -integral was introduced by Rice [9], Cherepanov [10], and Eshelby [11] to define the strength of the stress-strain field in nonlinear elastic 2-D crack problems. The  $J$ -integral was shown to be path-independent for nonlinear elastic and power-law hardening elastic-plastic materials. This path independence can be explained based on one singular point (crack tip) inside a closed contour in a singly connected domain.

In 3-D crack problems, the crack front forms a line singularity and the strength of the singularity ( $K$  or  $J$ ) could be varying all along the crack front. Therefore, the path independency is valid in a global sense; that is, the total (or average) strength of the singularity of the complete crack front is independent of the surface enclosing it. However, at a point on the crack front, the path independency is valid only over a small region around the crack front due to interacting singular fields at neighboring points on the crack front.

Consider a small tube of radius  $\epsilon$  around a segment of crack front of length  $\Delta$  as shown in Fig. 1a such that the limit of  $\Delta$  and  $\epsilon/\Delta$  tends to zero. The local  $J$ -integral, also referred to as the

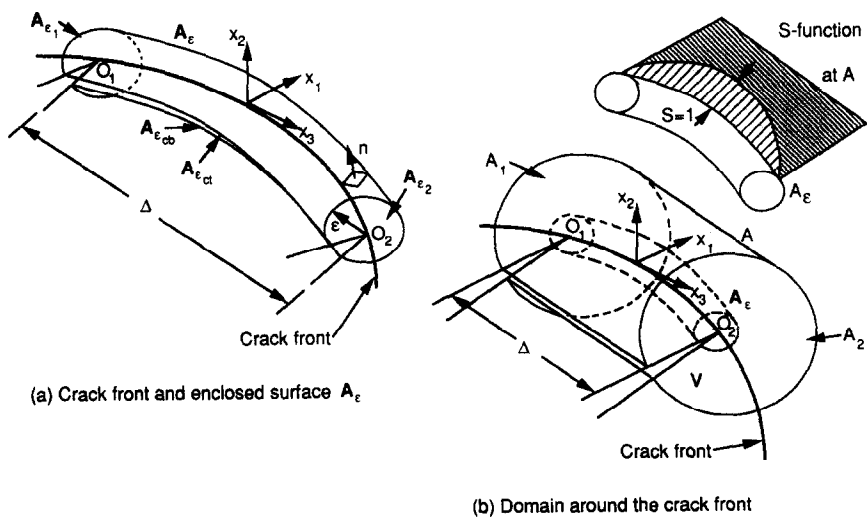


Fig. 1. Nomenclature and the domain description.

crack front integral, over the surface  $A_c$  is defined as (see Fig. 1a) [12, 20]

$$\int_{\Delta} J_{x_k} dx_3 = \lim_{\substack{c/\Delta \rightarrow 0 \\ \Delta \rightarrow 0}} \int \left[ W n_k - \sigma_{ij} \frac{\partial u_i}{\partial x_k} n_j \right] dA. \quad (1)$$

In eq. (1),  $W$  is the stress-work density,  $\sigma_{ij}$  is the stress tensor,  $u_i$  is the displacement vector, and  $n_k$  is the  $k$ th directional component of the unit normal vector on the closed surface  $A_c$ . The indices  $i$  and  $j$  take the values 1, 2, and 3, and  $k$  takes the values 1 and 2. Thus the local value of  $J_{x_k}$  is the total energy flux leaving the closed surface  $A_c$  per unit crack front length in the  $k$ th direction.  $J_{x_k}$  can be defined in any coordinate system, but the local crack front coordinate system  $x_1$ ,  $x_2$ , and  $x_3$  is convenient for crack-extension studies. Note that the axes  $x_1$  and  $x_3$  are in the crack plane and are normal and tangential to the crack front, respectively, while  $x_2$  is normal to the crack plane.

The complete surface integral, in terms of surfaces identified in Fig. 1a, is written as (henceforth the limits are dropped for convenience of presentation)

$$\int_{\Delta} J_{x_k} dx_3 = \int_{A_c} Q dA + \int_{A_{c_1} + A_{c_2}} Q dA + \int_{A_{c_{ct}} + A_{c_{cb}}} Q dA, \quad (2)$$

where

$$Q = \left[ W n_k - \sigma_{ij} \frac{\partial u_i}{\partial x_k} n_j \right] \quad (3)$$

$$W = \int_0^{\epsilon_{ij}} \sigma_{ij} d\epsilon_{ij}. \quad (4)$$

In eq. (2),  $A_{c_1}$  and  $A_{c_2}$  are cross-sectional areas of the tube at  $O_1$  and  $O_2$ , respectively. The subscripts  $ct$  and  $cb$  represent top and bottom crack surfaces, respectively. The stress-work density in eq. (4) is calculated over the complete strain path. The total strain tensor  $\epsilon_{ij}$  includes elastic, plastic, and thermal components. For linear elastic problems,  $W = (\sigma_{ij}\epsilon_{ij})/2$ . Note that eq. (2) involves only stress, displacement and strain fields but no material properties. However, to calculate stresses from strains the appropriate constitutive relationships (isotropic or anisotropic) must be used. Thus eq. (2) is applicable for general thermo-elastic-plastic and anisotropic material problems.

Fracture modes in a 3-D cracked solid can be represented by three modes: opening (mode I), shearing (mode II), and tearing (mode III) modes. The corresponding three modes of the  $J$ -integral are  $J_I$ ,  $J_{II}$ , and  $J_{III}$ . In eqs (1) or (2),  $J_{x_1}$  and  $J_{x_2}$  represent the total  $J$ -integral ( $J_I + J_{II} + J_{III}$ ) and the product integral  $[-2\sqrt{(J_I \cdot J_{II})}]$ , respectively [12, 19, 20]. Since the meaning of  $J_{x_3}$  integral is not clear for crack problems, it is not defined by eqs (1) or (2). However, the mode III integral is separately defined as [20]

$$\int_{\Delta} J_{III} dx_3 = \lim_{\substack{c/\Delta \rightarrow 0 \\ \Delta \rightarrow 0}} \left[ \int_{A_c} Q_3 dA + \int_{A_{c_1} + A_{c_2}} Q_3 dA + \int_{A_{c_{ct}} + A_{c_{cb}}} Q_3 dA \right], \quad (5)$$

where

$$Q_3 = W^{III} n_1 - \sigma_{3j} \frac{\partial u_3}{\partial x_1} n_j \quad (6)$$

$$W^{III} = \int_0^{\epsilon_{3j}} \sigma_{3j} d\epsilon_{3j}. \quad (7)$$

The indices  $i$  and  $j$  take values 1, 2, and 3. Equations (2) through (7) completely define all three  $J$ -integrals. As previously mentioned, the evaluation of surface integrals, eqs (2) and (5), is tedious and could introduce errors due to numerical integration of singular terms. Therefore, an alternate form of evaluating the above surface integrals, the equivalent domain integral (EDI) is presented in the next section.

#### Equivalent domain integral

Consider two tubular surfaces  $A_c$  and  $A$  spanning between two points  $O_1$  and  $O_2$  on the crack front (see Fig. 1b). The tube  $A$  is arbitrary and encloses the tube  $A_c$  on which the  $J$ -integral is evaluated. The surface integrals [eqs (2) and (5)] on  $A_c$  are converted into volume integrals by mathematical manipulations. First, the right hand sides of eqs (2) and (5) are multiplied by unity.

Next, from the resulting right hand side of the equation, subtract the product of the same integral over the surface  $A$  and zero, as shown below.

$$\int_{\Delta} J_{x_k} dx_3 = 1 \cdot \int_{A_c} Q dA + 1 \cdot \int_{A_{c_1} + A_{c_2}} Q dA + 1 \cdot \int_{A_{c_{cl}} + A_{c_{cb}}} Q dA - 0 \cdot \int_A Q dA \quad (8)$$

$$\int_{\Delta} J_{III} dx_3 = 1 \cdot \int_{A_c} Q_3 dA + 1 \cdot \int_{A_{c_1} + A_{c_2}} Q_3 dA + 1 \cdot \int_{A_{c_{cl}} + A_{c_{cb}}} Q_3 dA - 0 \cdot \int_A Q_3 dA. \quad (9)$$

Note that eqs (8) and (9) assume a unit extension of the crack front segment in the  $x_1$ -direction. Instead, if an arbitrary (nonuniform) virtual extension of the crack front is made, eqs (8) and (9) need to be modified to account for this variation. Therefore, an arbitrary but continuous function  $S(x_1, x_2, x_3)$  is introduced [19, 20, 23] that has the property  $S(x_1, x_2, x_3) = 0$  on the surface  $A$  and  $S(x_1, x_2, x_3) = S(x_3)$  on the surface  $A_c$ . Using the  $S$ -function, eqs (8) and (9) are rewritten as follows.

$$\int_{\Delta} J_{x_k} S dx_3 = \int_{A_c} QS dA + \int_{A_{c_1} + A_{c_2}} QS dA + \int_{A_{c_{cl}} + A_{c_{cb}}} QS dA - \int_A QS dA \quad (8a)$$

$$\int_{\Delta} J_{III} S dx_3 = \int_{A_c} Q_3 S dA + \int_{A_{c_1} + A_{c_2}} Q_3 S dA + \int_{A_{c_{cl}} + A_{c_{cb}}} Q_3 S dA - \int_A Q_3 S dA. \quad (9a)$$

As previously mentioned,  $\Delta$  is small ( $\lim \Delta \rightarrow 0$ ), and hence  $J_{x_k}$  is assumed to be constant over the crack front segment length  $\Delta$ . Then, eqs (8a) and (9a) are simplified to

$$J_{x_k} \cdot f = \int_{A_c} QS dA + \int_{A_{c_1} + A_{c_2}} QS dA + \int_{A_{c_{cl}} + A_{c_{cb}}} QS dA - \int_A QS dA \quad (10)$$

$$J_{III} \cdot f = \int_{A_c} Q_3 S dA + \int_{A_{c_1} + A_{c_2}} Q_3 S dA + \int_{A_{c_{cl}} + A_{c_{cb}}} Q_3 S dA - \int_A Q_3 S dA, \quad (11)$$

where

$$f = \int_{O_1}^{O_2} S(x_3) dx_3. \quad (12)$$

The parameter  $f$  is equivalent to the new crack surface area created by translating the crack front by  $S(x_3)$  in the  $x_1$ -direction. Evaluation of  $f$  in eq. (12) and the choice of the  $S$ -function will be discussed later. Equations (10) and (11) were further simplified by selecting the  $S$ -function such that the function has zero values at two end surfaces ( $O_1$  and  $O_2$ ) of the tubes  $A_c$  and  $A$  and non-zero between these two end faces. With this choice, the second surface integrals in eqs (10) and (11) become identically zero.

Now the integrals on the crack faces between the inner ( $A_c$ ) and outer ( $A$ ) tubes are added and subtracted from the right hand sides of eqs (10) and (11). This manipulation is performed to obtain the integrals on a closed surface that encloses a volume. After some elementary algebraic operations, eqs (10) and (11) are rewritten as follows.

$$J_{x_k} \cdot f = - \int_{A + (A - A_c)_{cl} + A_c + (A_c - A)_{cb}} QS dA + \int_{(A - A_c)_{cl}} QS dA + \int_{(A_c - A)_{cb}} QS dA + \int_{A_{c_{cl}} + A_{c_{cb}}} Q dA \quad (13)$$

$$J_{III} \cdot f = - \int_{A + (A - A_c)_{cl} + A_c + (A_c - A)_{cb}} Q_3 S dA + \int_{(A - A_c)_{cl}} Q_3 S dA + \int_{(A_c - A)_{cb}} Q_3 S dA + \int_{A_{c_{cl}} + A_{c_{cb}}} Q_3 dA. \quad (14)$$

The first term in eqs (13) and (14) is negative because the direction of integration on the inner surface of the tube ( $A_c$ ) is reversed. In eqs (13) and (14),  $(A - A_c)_{ct}$  and  $(A_c - A)_{cb}$  are the top and bottom crack surface areas between the two tubes  $A$  and  $A_c$ . The first term in eqs (13) and (14) encloses the volume between the two tubes  $A$  and  $A_c$ , which is represented as  $(V - V_c)$ . The rest of the terms in the right hand sides of eqs (13) and (14) are integrals on the crack faces. Hence,  $J_{x_k}$  and  $J_{III}$  are expressed as the sum of domain and crack surface integrals as follows.

$$J_{x_k} \cdot f = (J_{x_k} \cdot f)_{\text{domain}} + (J_{x_k} \cdot f)_{\text{crack faces}} \quad (15)$$

$$J_{III} \cdot f = (J_{III} \cdot f)_{\text{domain}} + (J_{III} \cdot f)_{\text{crack faces}} \quad (16)$$

Reference [20] obtained similar equations but did not include the crack face integrals. However, as will be shown later, for special cases the crack-face integrals vanish and the complete  $J$ -integral is given by the domain part of eqs (15) and (16).

*Domain integral.* Invoking Green's divergence theorem, the closed surface integrals of eqs (13) and (14) are written as a domain integral as follows.

$$\begin{aligned} (J_{x_k} \cdot f)_{\text{domain}} &= - \int Q S \, dA = - \int \left[ W n_k - \sigma_{ij} \frac{\partial u_i}{\partial x_k} n_j \right] S \, dA \\ &= - \int_{(V - V_c)} \left[ \frac{\partial (WS)}{\partial x_k} - \frac{\partial}{\partial x_j} \left( \sigma_{ij} \frac{\partial u_i}{\partial x_k} S \right) \right] dV. \end{aligned} \quad (17)$$

Hence, the domain integral is

$$(J_{x_k} \cdot f)_{\text{domain}} = - \int_{(V - V_c)} \left[ W \frac{\partial S}{\partial x_k} - \sigma_{ij} \frac{\partial u_i}{\partial x_k} \frac{\partial S}{\partial x_j} \right] dV - \int_{(V - V_c)} \left[ \frac{\partial W}{\partial x_k} - \sigma_{ij} \frac{\partial \epsilon_{ij}}{\partial x_k} \right] S \, dV. \quad (18)$$

Similarly, one has

$$(J_{III} \cdot f)_{\text{domain}} = - \int_{(V - V_c)} \left[ W^{III} \frac{\partial S}{\partial x_1} - \sigma_{3j} \frac{\partial u_3}{\partial x_1} \frac{\partial S}{\partial x_j} \right] dV - \int_{(V - V_c)} \left[ \frac{\partial W^{III}}{\partial x_1} - \sigma_{3j} \frac{\partial}{\partial x_1} \left( \frac{\partial u_3}{\partial x_j} \right) \right] S \, dV. \quad (19)$$

In deriving eqs (18) and (19), the equations of equilibrium

$$\frac{\partial \sigma_{ij}}{\partial x_j} = 0$$

and the small deformation strain-displacement relationships

$$\epsilon_{ij} = \frac{1}{2} \left( \frac{\partial u_i}{\partial x_j} + \frac{\partial u_j}{\partial x_i} \right)$$

were used.

In conventional finite element analysis, the equations of equilibrium are not satisfied pointwise in the domain that is modeled. Numerical experimentation showed that the differences between including and not including the terms involving the equations of equilibrium are of the order of  $10^{-3}$  to  $10^{-4}$  of the integral values for several problems. Therefore, in writing eqs (18) and (19) the equations of equilibrium are assumed to be satisfied exactly.

The terms in brackets in the second integral in eqs (18) and (19) are pointwise equal to zero for a linear elastic material. These terms, however, are non-zero in elastic-plastic and thermal problems. Since the present formulation is for general thermo-elastic-plastic problems, such simplifications are not incorporated.

The domain integral eq. (18) is rewritten in a matrix form as

$$(J_{x_k} \cdot f)_{\text{domain}} = - \int_{(V-V_c)} \left[ W \frac{\partial S}{\partial x_k} - \{u'_{x_k}\}^T [\sigma] \{S'\} \right] dV - \int_{(V-V_c)} \left[ \frac{\partial W}{\partial x_k} - \{\sigma\}^T \{\epsilon'_{x_k}\} \right] S dV, \quad (20)$$

where

$$\begin{aligned} \{\sigma\}^T &= \{\sigma_{11} \ \sigma_{22} \ \sigma_{33} \ \sigma_{12} \ \sigma_{23} \ \sigma_{31}\} \\ \{\epsilon'_{x_k}\}^T &= \left\{ \frac{\partial \epsilon_{11}}{\partial x_k} \frac{\partial \epsilon_{22}}{\partial x_k} \frac{\partial \epsilon_{33}}{\partial x_k} 2 \frac{\partial \epsilon_{12}}{\partial x_k} 2 \frac{\partial \epsilon_{23}}{\partial x_k} 2 \frac{\partial \epsilon_{31}}{\partial x_k} \right\} \\ \{u'_{x_k}\}^T &= \left\{ \frac{\partial u_1}{\partial x_k} \frac{\partial u_2}{\partial x_k} \frac{\partial u_3}{\partial x_k} \right\} \\ [\sigma] &= \begin{bmatrix} \sigma_{11} & \sigma_{12} & \sigma_{31} \\ \sigma_{12} & \sigma_{22} & \sigma_{23} \\ \sigma_{31} & \sigma_{23} & \sigma_{33} \end{bmatrix} \\ \{S'\}^T &= \left\{ \frac{\partial S}{\partial x_1} \frac{\partial S}{\partial x_2} \frac{\partial S}{\partial x_3} \right\} \end{aligned} \quad (21)$$

and the stress-work density  $W$  is

$$W = \int_0^\epsilon [\sigma_{11} d\epsilon_{11} + \sigma_{22} d\epsilon_{22} + \sigma_{33} d\epsilon_{33} + 2\sigma_{12} d\epsilon_{12} + 2\sigma_{23} d\epsilon_{23} + 2\sigma_{31} d\epsilon_{31}].$$

Similarly, the mode III integral eq. (19) is written as

$$(J_{III} \cdot f)_{\text{domain}} = - \int_{(V-V_c)} \left[ W^{III} \frac{\partial S}{\partial x_k} - \frac{\partial u_3}{\partial x_1} \{\sigma_3\}^T \{S'\} \right] dV - \int_{(V-V_c)} \left[ \frac{\partial W^{III}}{\partial x_1} - \{\sigma_3\}^T \{u''_3\} \right] S dV, \quad (22)$$

where

$$\begin{aligned} \{\sigma_3\}^T &= \{\sigma_{31} \ \sigma_{32} \ \sigma_{33}\} \\ \{u''_3\}^T &= \left\{ \frac{\partial^2 u_3}{\partial x_1^2} \frac{\partial^2 u_3}{\partial x_1 \partial x_2} \frac{\partial^2 u_3}{\partial x_1 \partial x_3} \right\} \end{aligned}$$

and

$$W^{III} = \int_0^\epsilon [\sigma_{33} d\epsilon_{33} + \sigma_{31} d\epsilon_{31} + \sigma_{32} d\epsilon_{32}]. \quad (23)$$

The numerical implementation of eqs (20) and (22) in an FE analysis with isoparametric elements is presented in the Appendix.

The integral  $J_{x_1}$  in eq (20) for the linear elastic case is equivalent to the total strain-energy release rate calculated by the virtual crack extension method [1–4, 13, 24, 25].

*Crack-face integrals.* The crack-face integrals in eqs (13) and (14) are

$$(J_{x_k} \cdot f)_{\text{crack face}} = \int_{(A-A_{ct})_{ct}} Q S dA + \int_{(A_c-A)_{cb}} Q S dA + \int_{A_{ct}+A_{cb}} Q dA \quad (24)$$

and

$$(J_{III} \cdot f)_{\text{crack face}} = \int_{(A-A_{ct})_{ct}} Q_3 S dA + \int_{(A_c-A)_{cb}} Q_3 S dA + \int_{A_{ct}+A_{cb}} Q_3 dA. \quad (25)$$

When the terms  $Q$  [eq. (3)] and  $Q_3$  [eq. (6)] are zero on the crack faces, obviously, the integrals in eqs (24) and (25) vanish. On the crack faces,  $n_1$  and  $n_3$  are always zero, while  $n_2 = -1$  on the top face ( $ct$ ) and  $n_2 = 1$  on the bottom face ( $cb$ ). Imposition of these conditions in eqs (3) and (6) results in the following.

For  $k = 1$

$$\begin{aligned}(Q)_{\text{crack face}} &= -\sigma_{i2} \frac{\partial u_i}{\partial x_1} n_2 \\ (Q_3)_{\text{crack face}} &= -\sigma_{32} \frac{\partial u_3}{\partial x_1} n_2.\end{aligned}\quad (26)$$

For  $k = 2$

$$(Q)_{\text{crack face}} = W n_2 - \sigma_{i2} \frac{\partial u_i}{\partial x_2} n_2. \quad (27)$$

Note that for  $k = 2$ ,  $Q_3$  is zero.

Thus for traction free crack faces the terms  $(J_{x_1})_{\text{crack face}}$  and  $(J_{\text{III}})_{\text{crack face}}$  vanish. In contrast, the term  $(J_{x_2})_{\text{crack face}}$  is no longer zero. As noted in ref. [23], the  $(J_{x_2})_{\text{crack face}}$  is zero only for pure mode I fracture problems or for a singular stress field alone. However, in any finite size cracked body the stress field consists of both singular and non-singular terms and, hence, the  $(J_{x_2})_{\text{crack face}}$  integral is not zero and cannot be neglected. The numerical evaluation of crack-face integrals involves the computation of singular integral terms, which are computationally cumbersome and the source of errors [23].

#### *Separation of modes in mixed-mode problems*

There are three modes of deformations in a cracked body, namely the opening mode (mode I), the shearing mode (mode II), and the tearing mode (mode III). The direct and decomposition methods are used to separate the mixed-mode fracture mechanics parameters into the three individual modes.

*Direct method.* The three components of the  $J$ -integral, namely,  $J_I$ ,  $J_{\text{II}}$  and  $J_{\text{III}}$ , are calculated from  $J_{x_1}$  and  $J_{x_2}$  of eq. (15) and  $J_{\text{III}}$  of eq. (16). Since  $J_{\text{III}}$  is directly calculated, the other two are calculated by solving the equations

$$\begin{aligned}J_{x_1} &= J_I + J_{\text{II}} + J_{\text{III}} \\ J_{x_2} &= -2\sqrt{(J_I J_{\text{II}})}.\end{aligned}\quad (28)$$

Equation (28) was used to obtain  $J_I$  and  $J_{\text{II}}$  as

$$\begin{aligned}J_I &= \frac{1}{4}[\sqrt{(J_{x_1} - J_{x_2} - J_{\text{III}})} + \sqrt{(J_{x_1} + J_{x_2} - J_{\text{III}})}]^2 \\ J_{\text{II}} &= \frac{1}{4}[\sqrt{(J_{x_1} - J_{x_2} - J_{\text{III}})} - \sqrt{(J_{x_1} + J_{x_2} - J_{\text{III}})}]^2.\end{aligned}\quad (29)$$

Thus, in a general mixed-mode crack problem, computation of  $J_{x_1}$  and  $J_{x_2}$  from eq. (13),  $J_{\text{III}}$  from eq. (14), and the use of eq. (29) completely defines all three modes of the  $J$ -integral. This procedure appears simple but the evaluation of  $J_{x_2}$  could be complicated and erroneous due to the numerical integration of singular functions. Furthermore, as explained in ref. [23], the local crack-face displacements are needed to distinguish between the opening and sliding (shearing) modes of deformation. Because of these reasons, separation of modes using eq. (29) may not be the best choice. Hence, an alternate method that avoids the evaluation of  $J_{x_2}$ , called the decomposition method, is used.

*Decomposition method.* The advantage of transforming the surface integral into a domain integral appears to be lost because of the non-zero crack-face integrals as shown in eqs (15), (16) and (27). These crack-face integrals are necessary to account for the terms containing the product of the singular and non-singular stress (strain) fields in the stress-work density expression. It was shown in ref. [23] that the product terms are eliminated by decomposing the stress and displacement fields into symmetric and antisymmetric parts. The resulting equation contains only the domain integrals. Hence, the method is attractive and is computationally efficient. The decomposition method, however, requires additional effort to create a symmetric mesh about the crack plane.

The decomposition of displacement and stress fields is straightforward for 2-D problems [19, 23, 26–28], but is slightly more complicated for 3-D problems [20]. Hence, the decomposition of displacement and stress fields corresponding to the three modes of fracture is presented.

Consider any two points  $P(x_1, x_2, x_3)$  and  $P'(x_1, -x_2, x_3)$  that are in the immediate neighborhood of the crack front and are symmetric about the crack plane as shown in Fig. 2. For any arbitrarily general deformation, the displacements and stresses at points  $P$  and  $P'$  can be expressed as a combination of symmetric and antisymmetric components as shown below.

$$\begin{Bmatrix} u_{1P} \\ u_{2P} \\ u_{3P} \end{Bmatrix} = \begin{Bmatrix} u_{1S} \\ u_{2S} \\ u_{3S} \end{Bmatrix} + \begin{Bmatrix} u_{1AS} \\ u_{2AS} \\ u_{3AS} \end{Bmatrix} \quad (30)$$

and

$$\begin{Bmatrix} u_{1P'} \\ u_{2P'} \\ u_{3P'} \end{Bmatrix} = \begin{Bmatrix} u_{1S} \\ -u_{2S} \\ u_{3S} \end{Bmatrix} + \begin{Bmatrix} -u_{1AS} \\ u_{2AS} \\ -u_{3AS} \end{Bmatrix}, \quad (31)$$

where subscripts  $S$  and  $AS$  denote the symmetric and antisymmetric components, respectively.

Equations (30) and (31) are used to determine the symmetric and antisymmetric displacements in terms of the displacements at points  $P$  and  $P'$  as

$$\begin{Bmatrix} u_1 \\ u_2 \\ u_3 \end{Bmatrix}_S = \frac{1}{2} \begin{Bmatrix} u_{1P} + u_{1P'} \\ u_{2P} - u_{2P'} \\ u_{3P} + u_{3P'} \end{Bmatrix} \quad (32)$$

$$\begin{Bmatrix} u_1 \\ u_2 \\ u_3 \end{Bmatrix}_{AS} = \frac{1}{2} \begin{Bmatrix} u_{1P} - u_{1P'} \\ u_{2P} + u_{2P'} \\ u_{3P} - u_{3P'} \end{Bmatrix}.$$

Similarly, the symmetric and antisymmetric components of the stresses are expressed in terms of the stresses at points  $P$  and  $P'$  (see Fig. 3) as

$$\begin{Bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{33} \\ \sigma_{12} \\ \sigma_{23} \\ \sigma_{31} \end{Bmatrix}_S = \frac{1}{2} \begin{Bmatrix} \sigma_{11P} + \sigma_{11P'} \\ \sigma_{22P} + \sigma_{22P'} \\ \sigma_{33P} + \sigma_{33P'} \\ \sigma_{12P} - \sigma_{12P'} \\ \sigma_{23P} - \sigma_{23P'} \\ \sigma_{31P} + \sigma_{31P'} \end{Bmatrix} \quad (33)$$

$$\begin{Bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{33} \\ \sigma_{12} \\ \sigma_{23} \\ \sigma_{31} \end{Bmatrix}_{AS} = \frac{1}{2} \begin{Bmatrix} \sigma_{11P} - \sigma_{11P'} \\ \sigma_{22P} - \sigma_{22P'} \\ \sigma_{33P} - \sigma_{33P'} \\ \sigma_{12P} + \sigma_{12P'} \\ \sigma_{23P} + \sigma_{23P'} \\ \sigma_{31P} - \sigma_{31P'} \end{Bmatrix}.$$

The symmetric and antisymmetric displacement and stress fields are further separated into mode I, mode II, and mode III components as follows.

$$\{u\} = \{u\}^I + \{u\}^{II} + \{u\}^{III} = \frac{1}{2} \begin{Bmatrix} u_{1P} + u_{1P'} \\ u_{2P} - u_{2P'} \\ u_{3P} + u_{3P'} \end{Bmatrix} + \frac{1}{2} \begin{Bmatrix} u_{1P} - u_{1P'} \\ u_{2P} + u_{2P'} \\ 0 \end{Bmatrix} + \frac{1}{2} \begin{Bmatrix} 0 \\ 0 \\ u_{3P} - u_{3P'} \end{Bmatrix} \quad (34)$$



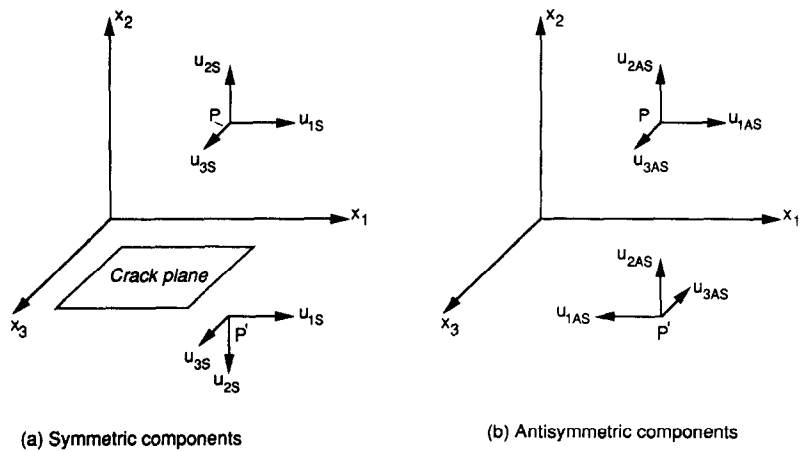


Fig. 2. Symmetric and antisymmetric displacement components.

and

$$\{\sigma\} = \{\sigma\}^I + \{\sigma\}^{II} + \{\sigma\}^{III} = \frac{1}{2} \begin{Bmatrix} s_{11p} + \sigma_{11p'} \\ \sigma_{22p} + \sigma_{22p'} \\ \sigma_{33p} + \sigma_{33p'} \\ \sigma_{12p} - \sigma_{12p'} \\ \sigma_{23p} - \sigma_{23p'} \\ \sigma_{31p} + \sigma_{31p'} \end{Bmatrix} + \frac{1}{2} \begin{Bmatrix} \sigma_{11p} - \sigma_{11p'} \\ \sigma_{22p} - \sigma_{22p'} \\ 0 \\ \sigma_{12p} + \sigma_{12p'} \\ 0 \\ 0 \end{Bmatrix} + \frac{1}{2} \begin{Bmatrix} 0 \\ 0 \\ \sigma_{33p} - \sigma_{33p'} \\ 0 \\ \sigma_{23p} + \sigma_{23p'} \\ \sigma_{31p} - \sigma_{31p'} \end{Bmatrix}. \quad (35)$$

Similar equations in ref. [20] had typographical errors. The mode I, mode II, and mode III displacements [eq. (34)] and stresses [eq. (35)] are used to directly evaluate  $J_I$ ,  $J_{II}$ , and  $J_{III}$  from  $J_{x_1}$  using eq. (15). Note that the surface integral in eq. (15) is required only when the crack face is loaded. The  $J_{x_2}$  integral for each of these modes of deformation is identically equal to zero because of the orthogonality of the modes of deformations. Hence, the decomposition method involves only the evaluation of domain integrals and is computationally efficient.

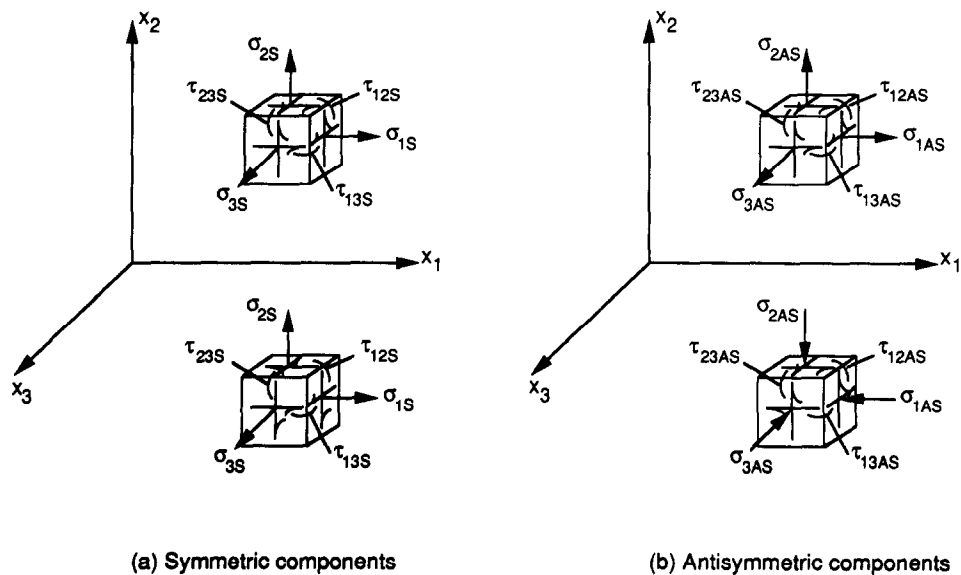


Fig. 3. Symmetric and antisymmetric stress components.

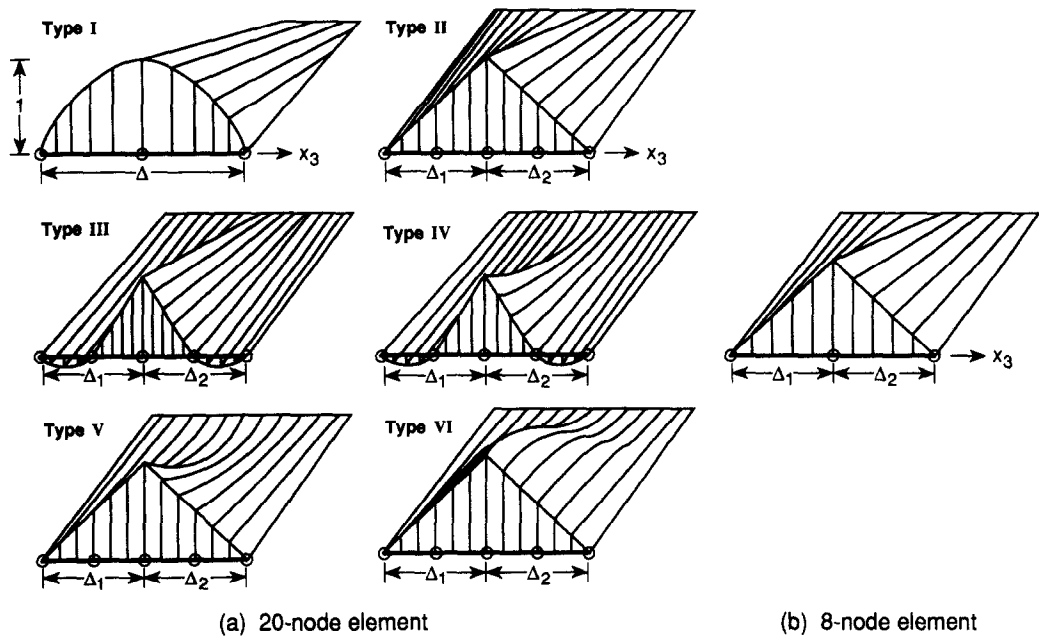


Fig. 4. Definition of *S*-functions for 20-node and eight-node isoparametric finite-elements.

*S*-functions

As mentioned previously, the *S*-function is an arbitrary but continuous function with a zero value on the surface *A* and at the ends of the tube (*A*<sub>1</sub> and *A*<sub>2</sub>) and a non-zero value (varying between zero and one) on the surface *A*<sub>ε</sub> (see Fig. 1). On the tube surface *A*<sub>ε</sub>, the *S*-function is a function of only *x*<sub>3</sub> and has a value of one at the location where the *J*-integral is required. The *S*-functions are conveniently defined by specifying the values of *S* at the nodes and using the element shape functions (see Appendix). Figure 4 presents several types of *S*-functions for domains spanning one or two elements in the *x*<sub>3</sub>-direction. Table 1 presents the values of *f* [see eq. (12)] that correspond to each of the *S*-functions shown in Fig. 4. Also, the *S*-functions for both eight-noded and 20-noded isoparametric elements are presented. For the eight-noded element, a linear *S*-function is defined, while for 20-noded elements several combinations of linear and quadratic functions are defined. Note that the values for *f* depend only on the variation of the *S*-function in the *x*<sub>3</sub>-direction.

The *J*-integrals for various cracked 3-D solids were calculated using the various *S*-functions presented in Fig. 4. The results of these numerical experiments will be discussed later.

RESULTS AND DISCUSSION

The EDI method was applied to several pure mode-I, mode-II, mode-III, and mixed-mode fracture problems to evaluate the accuracy, domain independency, and *S*-function independency.

Table 1. *S*-functions and *f*-integrals

<i>S</i> -function type	<i>S</i> -function		<i>f</i> -integral
	Left segment $\zeta = x_3/\Delta_1$	Right segment $\zeta = x_3/\Delta_2$	
I	$(1 - \zeta^2)$	$(1 - \zeta^2)$	$2\Delta/3$
II	$(1 + \zeta)/2$	$(1 - \zeta)/2$	$(\Delta_1 + \Delta_2)/2$
III	$(\zeta^2 - \zeta)/2$	$(\zeta^2 + \zeta)/2$	$(\Delta_1 + \Delta_2)/6$
IV	$(\zeta^2 - \zeta)/2$	$(\zeta^2 + \zeta)/2$	$(\Delta_1 + \Delta_2)/6$
V	$(1 + \zeta)/2$	$(1 - \zeta)/2$	$(\Delta_1 + \Delta_2)/2$
VI	$(1 + \zeta)/2$	$(1 - \zeta)/2$	$(\Delta_1 + \Delta_2)/2$
Eight-noded element	$(1 + z)/2$	$(1 - \zeta)/2$	$(\Delta_1 + \Delta_2)/2$

Although the method formulated above is for general anisotropic and nonlinear materials, the results presented here are restricted to linear elastic and isotropic materials with Poisson's ratio of 0.3. Throughout the analysis only *non-singular* elements were used around the crack front. Wherever possible, a *rectangular* arrangement of finite elements was used near and around the crack front to evaluate the accuracy of the rectangular type of modeling.

First, the EDI method was applied to a 3-D cracked body subjected to a combination of known mode-I, mode-II, and mode-III singular stress fields. Then, the method was applied to various finite size crack problems subjected to loadings that produce single or mixed-mode deformations. The specimen configurations considered were the middle-crack tension specimen and embedded cracks in circular cylindrical rods subjected to tension, torsion, and shearing loads. The computed  $J$ -integral values are compared with those from literature wherever possible. Both the direct and decomposition methods were used to separate the mode-III component. Only the decomposition method was used to separate mode-I and mode-II components because of the singular integration involved in the direct method [23]. All of the above analyses used FE models with 20-node isoparametric elements. Typical results are also presented for models with eight-node isoparametric elements.

#### *Singular field loading on a cracked body*

Consider a single-edge-cracked specimen with a straight crack front as shown in Fig. 5a. This crack configuration was subjected to mode-I, mode-II or mode-III, or a combination of these modes. This problem demonstrated the accuracy of the method without introducing the non-singular stress field that occurs in any FE analysis of a cracked body. The displacements in each of the three modes of deformation are given in terms of the spherical coordinates ( $r$ ,  $\theta$ , and  $z$ ) [29] as follows.

Mode-I displacements:

$$\begin{aligned} u_1 &= \frac{K_I}{G} \sqrt{\left(\frac{r}{2\pi}\right)} \cos \frac{\theta}{2} \left[ 1 - 2\nu + \sin^2 \frac{\theta}{2} \right] \\ u_2 &= \frac{K_I}{G} \sqrt{\left(\frac{r}{2\pi}\right)} \sin \frac{\theta}{2} \left[ 2 - 2\nu - \cos^2 \frac{\theta}{2} \right] \\ u_3 &= 0. \end{aligned} \quad (36)$$

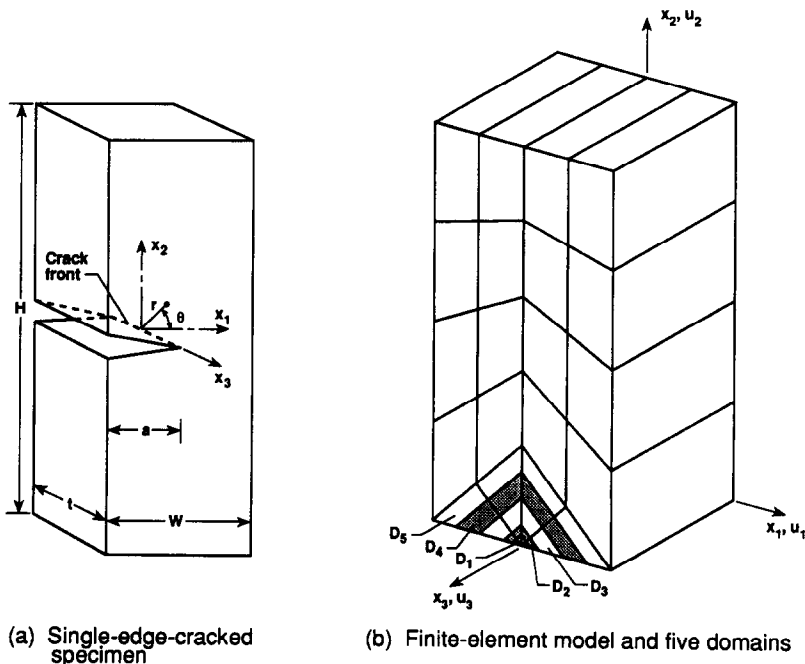


Fig. 5. Specimen and the finite-element model.

Mode-II displacements:

$$\begin{aligned} u_1 &= \frac{K_{II}}{G} \sqrt{\left(\frac{r}{2\pi}\right)} \sin \frac{\theta}{2} \left[ 2 - 2\nu + \cos^2 \frac{\theta}{2} \right] \\ u_2 &= \frac{K_{II}}{G} \sqrt{\left(\frac{r}{2\pi}\right)} \cos \frac{\theta}{2} \left[ -1 + 2\nu - \sin^2 \frac{\theta}{2} \right] \\ u_3 &= 0. \end{aligned}$$

(37)

Mode-III displacements:

$$\begin{aligned} u_1 &= 0 \\ u_2 &= 0 \\ u_3 &= \frac{K_{III}}{G} \sqrt{\left(\frac{2r}{\pi}\right)} \sin \frac{\theta}{2}, \end{aligned}$$

(38)

where  $G = E/(2(1 + \nu))$ .

The above displacement fields were imposed on the cracked body shown in Fig. 5a. The resulting stresses and displacements in the solid were used to calculate the  $J$ -integrals by the EDI method.

Utilizing the symmetries in the problem, a representative quarter of the solid was modeled with 20-node isoparametric elements as shown in Fig. 5b. One layer of elements was used to model the entire thickness. The finite element model had 320 nodes and 36 elements. The displacements for each mode of deformations were calculated at each node of the model from eqs (36)–(38) and were used as input for the EDI algorithm.

*Domains and S-functions.* Five domains were used in the calculations. These domains are shown in Fig. 5b as  $D_1, D_2 \cdots D_5$ . Each domain consisted of one *ring* of four elements around the crack front. The surface that is nearest to the crack front of each domain corresponds to the surface  $A_c$ . Similarly, the surface that is farthest from the crack front of each domain corresponds to the surface  $A$ . (Thus, for example, the surface  $A$  for the domain  $D_2$  will be the surface  $A_c$  for domain  $D_3$ .) On the surfaces  $A$  of each domain, the  $S$ -function was prescribed to be zero. On the surfaces  $A_c$  several types of  $S$ -functions in the  $x_3$ -direction (the six types are shown in Fig. 4) were considered. Since one layer of elements was used to model the thickness of the solid,  $\Delta_2$  was set equal to zero for the type II through VI  $S$ -functions.

*J-integral results.* Table 2 presents the normalized values of  $J_{x_1}$  calculated for four domains ( $D_2$  to  $D_5$ ) and six types of  $S$ -functions. The imposed displacement field on the model corresponds to  $K_I = 1$ . The  $J_{II}$ -integral, as expected, was computed to the order of machine zero and hence is not shown here. Results for all four domains and type I, II, III, and V  $S$ -functions are in excellent agreement with the exact solution. Type IV and VI  $S$ -functions are quadratic in the radial direction. Although these results are within 2% of the exact solution they are not as accurate as other types. Similar trends were observed when quadratic  $S$ -functions were used on several other crack problems. These results suggest that the simple type I  $S$ -function will give accurate  $J$ -integrals.

The computed integrals were inaccurate when the domain  $D_1$  was used. The  $J_{x_1}$ -integrals for this domain were about 10–15% higher than those for other domains. This behavior is attributed to the errors in the numerical integration of the imposed singular stress field.

Table 2. Comparison of  $J_{x_1}$  for various  $S$ -functions and domains in a  $K_I = 1$  stress field loaded specimen

S-function type	$\sqrt{(J_{x_1} E / (1 - \nu^2))}$ Domains			
	$D_2$	$D_3$	$D_4$	$D_5$
I	1.0008	1.0005	1.0013	1.0018
II	1.0008	1.0005	1.0044	1.0018
III	1.0008	1.0005	1.0107	1.0018
IV	1.0144	1.0200	1.0085	1.0062
V	1.0053	1.0070	1.0037	1.0032
VI	0.9870	0.9870	1.0129	0.9973

Table 3. Comparison of  $J$ -integral from EDI method with exact solutions for mixed-mode singular field loadings (type I  $S$ -functions)

$K_I$	Loading $K_{II}$ $K_{III}$		$J_{x_1}$				
			Exact	$D_2$	EDI method		
					$D_3$	$D_4$	$D_5$
1	0	0	0.9100	0.9107	0.9105	0.9112	0.9116
0	1	0	0.9100	0.9142	0.9156	0.9127	0.9121
0	0	1	1.3000	1.3015	1.3020	1.3011	1.3010
1	1	0	1.8200	1.7929	1.7864	1.8035	1.8096
0	1	1	2.2100	2.2158	2.2176	2.2138	2.2131
1	0	1	2.2100	2.2122	2.2125	2.2123	2.2126
1	1	1	3.1200	3.0945	3.0885	3.1046	3.1106

Note that in the above analysis one ring of elements around the crack front was used in each domain. Multiple rings of elements in each domain (for example, combining the  $D_2$  and  $D_3$  domains) gave results identical to those with a single ring of elements. Multiple rings, however, increase the data preparation efforts considerably. Thus, domains with a minimum number of rings of elements (in this case one ring) are preferable.

The domain independency observed in this example is expected since this is a plane-strain problem where the strength of the singularity is constant along the crack front ( $x_3$ -axis). Therefore, for crack bodies having a constant singularity strength along the crack front, the  $J_{x_1}$ -integral is independent of the domain.

Table 3 presents the normalized  $J_{x_1}$ , calculated assuming several linear combinations of the  $K_I$ ,  $K_{II}$ , and  $K_{III}$  displacement fields given by eqs (36)–(38). Four domains, each with type I  $S$ -functions, were used. The  $J_{x_1}$  calculated for four domains from eq. (15) agrees very well with the exact solutions. The maximum difference is less than 2% for the  $K_I = K_{II} = 1$  loading in domain  $D_3$ .

#### Middle-crack tension specimen

A typical middle-crack tension,  $M(T)$ †, specimen of  $W/a = 2$ ,  $t/a = 1$ ,  $H/a = 8$ , and crack length  $a$  subjected to a uniform tension stress  $\sigma$  at  $x_2 = \pm H/2$  was analyzed. The FE mesh shown in Fig. 6 was used for the analysis by imposing symmetry conditions at  $x_1 = -a$ ,  $x_2 = 0$  (on the uncracked plane), and  $x_3 = 0$  planes. The model was comprised of six unequal layers with thicknesses  $0.22t$ ,  $0.13t$ ,  $0.08t$ ,  $0.04t$ ,  $0.02t$ , and  $0.01t$  as shown in Fig. 6a. (The layer with the smallest thickness is near the  $x_3 = t/2$  surface.) Five domains,  $D_1, D_2 \dots D_5$ , as shown in Fig. 6b, and type I  $S$ -function (linear in radial direction and quadratic in  $x_3$ -direction, see Table 1) were used to calculate the domain integrals.

*Plane-strain analysis.* First, the  $M(T)$  specimen was analyzed for a plane-strain condition by imposing  $u_3 = 0$  on  $x_3 = 0$  and  $x_3 = t/2$  planes. The  $J_{x_1}$  values were calculated for the five domains all along the crack front using each of the six layers in the thickness direction. The average normalized value of  $J_{x_1}$ ,  $J_{x_1} E / [\pi \sigma^2 a (1 - \nu^2)]$ , was 1.410 with a maximum variation of less than 0.1%. This value agrees very well with the handbook value of 1.414 and the VCCT method [6] value of 1.424. These results also suggest that a simple linear  $S$ -function in the radial direction yields accurate  $J_{x_1}$  values.

*Three-dimensional analysis.* The  $M(T)$  specimen was reanalyzed by relaxing the plane-strain condition, i.e. by removing the boundary condition  $u_3 = 0$  on the  $x_3 = t/2$  plane. Three domain definitions,  $D_A$ ,  $D_B$ , and  $D_C$ , were used. The domain and  $S$ -function definitions for domain  $D_3$  are illustrated in Fig. 6b. In  $D_A$ , each ring of elements around the crack front represents a domain. In this case, the radius ( $\epsilon$ ) of the inner surface (the  $A_\epsilon$  surface on which the  $J_{x_1}$  is evaluated) was different for each domain; it varied from 0.0 to 0.734 times the crack length  $a$  for domains  $D_1$  to  $D_5$ , respectively. The other two domains ( $D_B$  and  $D_C$ ) involve a constant inner surface  $A_\epsilon$  and a variable outer surface. Domain  $D_B$  had  $\epsilon = a/10$  and, hence, the domain  $D_3$  included the second and third rings of elements around the crack front. The corresponding  $S$ -function definition in the

†ASTM abbreviation for middle-crack tension.

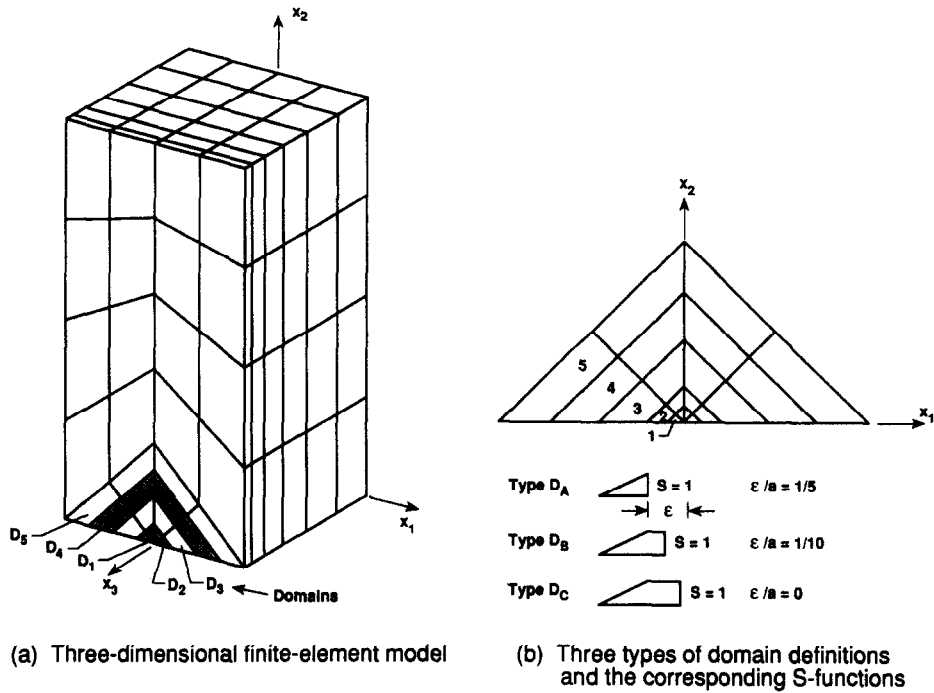


Fig. 6. Three-dimensional finite-element modeling of  $M(T)$  specimen.

radial direction is bilinear, as shown in Fig. 6b. The domain  $D_C$  had  $\epsilon = 0$ . Therefore, for example, domain  $D_3$  included first, second, and third rings of elements.

Table 4 presents the normalized  $J_{x_1}$  for all five domains using the three domain definitions. As expected,  $J_{x_1}$  for the interior layer is independent of the domain and the domain definition. In contrast,  $J_{x_1}$  for the last layer with a  $D_A$  domain definition shows a strong domain dependency. As the radius (or mean distance from the crack front) of the inner surface of the domain becomes large, the stress, strain, and displacement fields on this surface also include the effect of the singular field from the other segments of the crack front. Evaluation of the  $J_{x_1}$ -integral on this surface would yield values which are different if the single crack front segment alone had contributed to the deformation field. If the inner surface of the domain is close to or at the crack front, the interaction from the neighboring segments of the crack is either very small or nonexistent. Hence, the  $D_B$  ( $\epsilon/a = 0.1$ ) and  $D_C$  domain definitions gave more accurate results than the type  $D_A$ . These results suggest that the radius of the inner surface should be less than or equal to one-tenth of the crack front. Furthermore, a domain consisting of only one ring of elements is sufficient to calculate accurate values of  $J$  provided the domain is very close to the crack front. Note that domains  $D_1$

Table 4. Comparison of normalized  $J$  calculated from various domain definitions for  $M(T)$  specimen ( $W/a = 4, H/a = 8, t/a = 1, \nu = 0.3$ )

		$EJ_{x_1}/\{\pi\sigma^2a(1-\nu^2)\}$			
		EDI method			VCCT method
		Domain type			
$z/t$	Domain	$D_A$	$D_B$	$D_C$	
0.11 (interior layer)	$D_1$	1.558	1.558	1.558	1.575
	$D_2$	1.551	1.551	1.571	
	$D_3$	1.544	1.559	1.579	
	$D_4$	1.547	1.572	1.592	
	$D_5$	1.543	1.582	1.602	
0.495 (exterior layer)	$D_1$	1.293	1.293	1.293	1.279
	$D_2$	1.380	1.380	1.296	
	$D_3$	1.468	1.389	1.305	
	$D_4$	1.498	1.400	1.316	
	$D_5$	1.526	1.411	1.327	

and  $D_2$  always satisfy the above conditions, hence, the results were accurate and agreed very well with the VCCT method [6].

The global average value of  $J_{x_1}$  over the crack front, however, is domain independent. Any ring of elements (even with a  $D_A$  domain definition) spanning the complete length of the crack front yields accurate values. For example, the global average values of  $J_{x_1}$ ,  $EJ_{x_1}/[\pi\sigma^2a(1-\nu^2)]$ , for the four domains were 1.521 ( $D_2$ ), 1.523 ( $D_3$ ), 1.529 ( $D_4$ ), and 1.531 ( $D_5$ ). Only the innermost domain was less accurate, 1.433 ( $D_1$ ), because of errors in the stress-strain fields very near the crack front. In the above analyses a  $D_A$  domain definition was used and each ring had 24 elements.

Figure 7 compares the normalized  $J_{x_1}$  along the crack front from EDI and VCCT [6] methods. Excellent agreement is observed between the two methods. Plane-strain results are also shown in the figure as a reference solution.

#### Embedded penny-shaped crack in circular rod

An embedded penny-shaped crack of radius  $a$  in a circular cylindrical rod with  $D/a = 10$  and  $H/a = 40$  is shown in Fig. 8. Two types of loadings, uniform tension and torsion, were considered. Note that  $\bar{x}_1$ ,  $\bar{x}_2$ , and  $\bar{x}_3$  represent the global coordinate system and  $\bar{u}_1$ ,  $\bar{u}_2$  and  $\bar{u}_3$  represent the corresponding displacements. Utilizing the symmetry in the problem one-eighth of the specimen was modeled. Figure 8b shows the FE model of the lower eighth of the specimen. The model has 5143 nodes and 1020 20-noded elements with six layers in the circumferential direction as shown in Fig. 8b. Figure 8c shows the details of the modeling near a point on the crack front and the two domains  $D_1$  and  $D_2$  used in EDI calculations. Note the rectangular arrangement of elements. This arrangement is in contrast to the polar arrangement used in previous examples and by other investigators [20]. Two domains and type I  $S$ -functions were used to evaluate the integrals.

**Remote tension load.** The solid is loaded by a uniform stress  $\sigma$  on the  $\bar{x}_3 = \pm H/2$  planes. The  $J_{x_1}$ -integrals were calculated for the two domains ( $D_A$  domain definition) in each of the six layers in the circumferential direction. The  $J_{x_1}$  values for all six layers calculated in either  $D_1$  or  $D_2$  were identical. This is expected because of the axisymmetric nature of the problem. The normalized  $EJ_{x_1}/[\sigma^2a(1-\nu^2)]$  values from the EDI method for domains  $D_1$  and  $D_2$  are 1.333 and 1.299, respectively. The exact analytical solution due to Sneddon [30] for a penny-shaped crack in an infinite solid is 1.273, while Benthem and Koiter's asymptotic solution [31] for a finite size solid is 1.275. Thus, the normalized  $J_{x_1}$  values from domains  $D_1$  and  $D_2$  differed by less than 3% from each other and are about 2–5% higher than the closed form solutions [30, 31]. The result for domain  $D_2$  is more accurate than that for domain  $D_1$ , about 2% higher than Benthem and Koiter's value. The larger error in the domain  $D_1$  may be attributed to the fact that only two elements were used around the crack front and the whole domain is within the singular field. A  $D_C$  domain definition

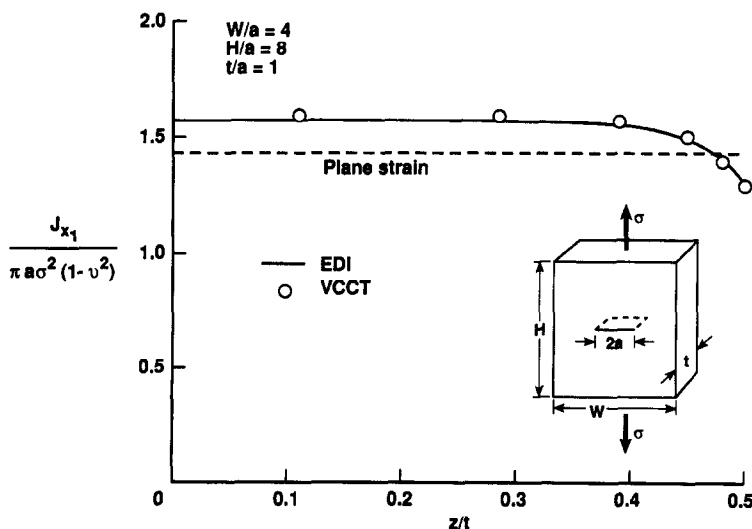


Fig. 7. Variation of normalized  $J_{x_1}$  along the crack front for an  $M(T)$  specimen.

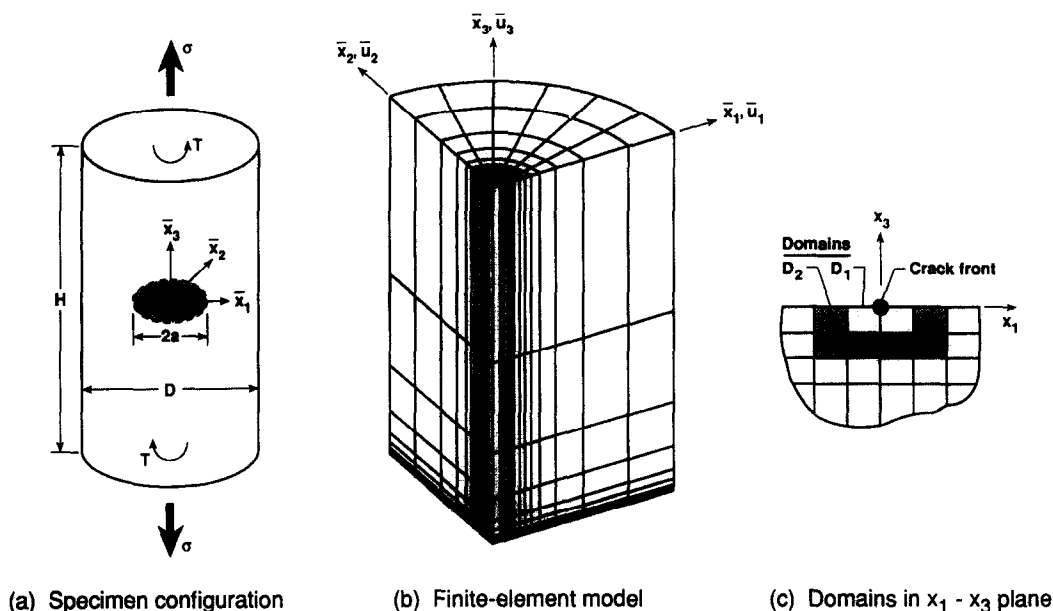


Fig. 8. Embedded penny-shaped crack in a circular rod subjected to tension and torsion loadings.

of domain  $D_2$ , which included the two rings of elements around the crack, also gave results identical to the  $D_2$  domain solution.

**Remote torsion loading.** A torque of magnitude  $T = \pi a^4 G/H$  was imposed at the two ends of the specimen ( $\bar{x}_3 = \pm H/2$ ), where  $G = E/[2(1 + \nu)]$ . This corresponds to an angular twist of magnitude  $2/H$  in the uncracked rod. The displacement field for this loading in the uncracked rod is

$$\begin{aligned}\bar{u}_1 &= -(2/H)\bar{x}_3\bar{x}_2 \\ \bar{u}_2 &= (2/H)\bar{x}_3\bar{x}_1 \\ \bar{u}_3 &= 0.\end{aligned}\quad (39)$$

The boundary conditions  $\bar{u}_1 = 0$  on  $\bar{x}_2 = 0$  plane,  $\bar{u}_2 = 0$  on  $\bar{x}_1 = 0$  plane, and  $\bar{u}_3 = 0$  at all nodes were imposed on the FE model. On the face  $\bar{x}_3 = -H/2$  the displacements given by eq. (39) were imposed on the model. The  $J$ -integral was calculated using both the direct and decomposition methods. The normalized values,  $EJ_{III}a^5/[(1 + \nu)T^2]$ , from the direct method for domains  $D_1$  and  $D_2$  are 0.2402 and 0.2389, and from the decomposition method for domains  $D_1$  and  $D_2$  are 0.2388 and 0.2374, respectively. The value of the normalized integral for an infinite solid obtained with the analytical solution by Lowengrub and Sneddon [32] was 0.2293. Since only mode-III loading was applied, the total integral  $J_{x_1}$  [eq. (15)] and the  $J_{III}$  integral [eq. (16)] are nearly identical for both domains. The value of  $J_{III}$  calculated by the EDI method is about 4% larger than Lowengrub and Sneddon's infinite body solution [32].

#### *Inclined embedded penny-shaped crack*

The EDI algorithm is next applied to problems involving mixed-mode deformations. Figure 9 shows an embedded inclined penny-shaped crack in a circular rod subjected to a uniform stress  $\sigma$ . The crack plane is inclined at an angle  $\alpha$  to the  $\bar{x}_1$ - $\bar{x}_2$  plane. The solution to this problem can be obtained as the sum of two solutions: the solution to a penny-shaped crack subjected to traction normal to the crack faces of the magnitude  $\sigma_n = (\sigma/2)(1 + \cos 2\alpha)$  and the solution of a penny-shaped crack subjected to shear traction on the crack faces of magnitude  $\tau = -(\sigma/2)\sin 2\alpha$ . The exact solutions for an infinite solid with the above mentioned traction are given by Cherepanov [12] for  $\nu = 0$  and Kassir and Sih [33] for a general value of Poisson's ratio  $\nu$ . The strain



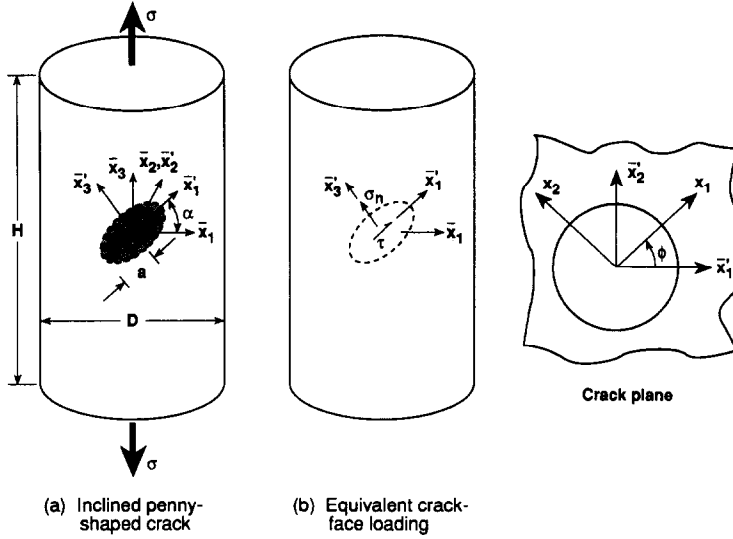


Fig. 9. Inclined penny-shaped crack embedded in a circular rod under tension  $\sigma$ .

energy release rates for the three modes, after converting the stress-intensity factors given in ref. [33] using plane-strain assumptions, are

$$\begin{aligned} G_I &= \frac{\sigma^2 a}{\pi E} (1 + \cos 2\alpha)^2 \\ G_{II} &= \frac{4(1 - \nu^2)a\sigma^2}{\pi(2 - \nu)^2 E} \sin^2 2\alpha \cos^2 \phi \\ G_{III} &= \frac{4(1 - \nu)(1 - \nu^2)a\sigma^2}{\pi(2 - \nu)^2 E} \sin^2 2\alpha \sin^2 \phi, \end{aligned} \quad (40)$$

where  $\phi$  is the angle measured from the  $\bar{x}'_1$ -axis on the crack plane (see Fig. 9b). These results for the infinite size solid are used to compare with the results from EDI method for a finite size rod.

A cylindrical rod with  $D/a = 5$  and  $H/a = 10$  was used in the analysis. Utilizing the symmetry in the problem one-half of the solid was modeled. The model has 9547 nodes and 2000 20-noded parabolic elements (see Fig. 10). Symmetry conditions ( $\bar{u}_2 = 0$ ) were imposed on the  $\bar{x}_2 = 0$  plane.

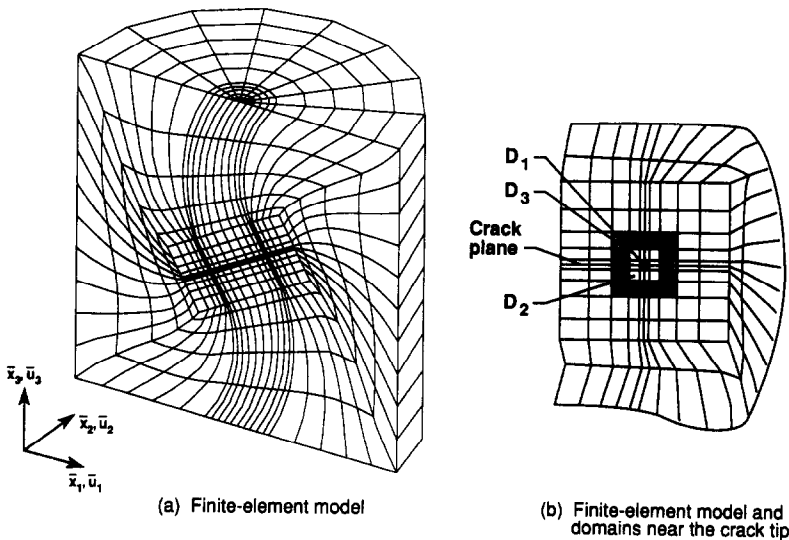


Fig. 10. Finite-element model for an inclined embedded penny-shaped crack.

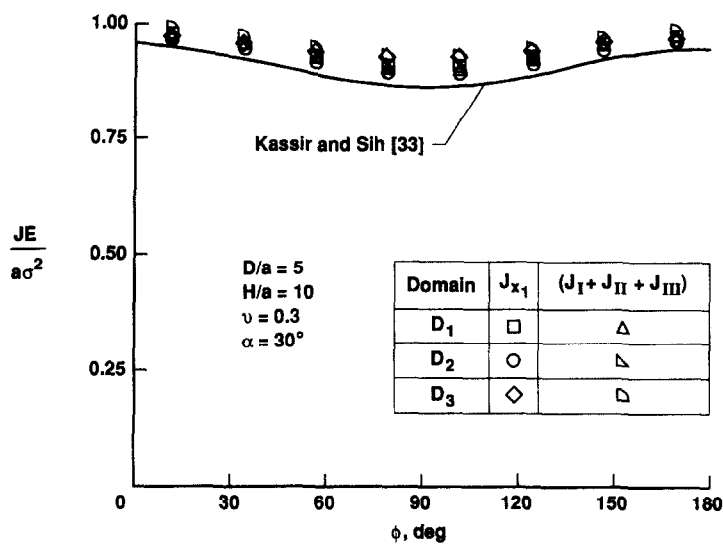


Fig. 11. Normalized  $J$ -distribution along the crack front of an embedded penny-shaped crack.

The three domains  $D_1$ ,  $D_2$ , and  $D_3$  used are shown in Fig. 10b. The normalized total  $J$ -integral all along the crack front calculated using eq. (15) ( $J_{x_1}$ ) and the total value of  $J$  (sum of  $J_I$ ,  $J_{II}$ , and  $J_{III}$ ) from the decomposition method are shown in Fig. 11. The Kassir and Sih [33] solution is shown by the solid line in this figure. The total values of  $J$  from the direct and the decomposition methods agree with each other for all three domains. EDI results from all three domains are about 2–7% larger than Kassir and Sih’s solution because a finite size solid was analyzed. Among the three domains, the domain  $D_2$  gave the lowest value while the domain  $D_3$  gave the largest. The maximum difference between the two domains  $D_2$  and  $D_3$  was about 3%.

Figure 12a shows the distribution of the normalized  $J_I$ ,  $J_{II}$  and  $J_{III}$  values along the crack front obtained from the decomposition method. The individual mode components from the EDI method agree well with those determined by Kassir and Sih all along the crack front. The small differences between the two solutions can be attributed to the finite solid. It is interesting to note that Cherepanov’s solutions (with  $\nu = 0$ ) for  $J_{II}$  and  $J_{III}$  are about 28% lower than Kassir and Sih’s solution (with  $\nu = 0.3$ ).

Figure 12b compares the  $J_{III}$  values calculated from the direct [eq. (16)] and decomposition methods with those from the Kassir and Sih solution [33]. For all three domains, the two EDI procedures agree well with each other and with Kassir and Sih.

*Inclined semi-circular surface crack in a tension rod*

A semi-circular prismatic rod with an inclined semi-circular surface crack was analyzed. The centrally located semi-circular surface crack plane is oriented at an angle of  $30^\circ$  to the  $\bar{x}_1$ -axis of the rod (see insert sketch in Fig. 13). The rod is subjected to a remote tension loading. The FE model of Fig. 10a was utilized to analyze this problem. The three domains  $D_1$ ,  $D_2$ , and  $D_3$  shown in Fig. 10b were used in the EDI calculation. The total value of  $J(J_{x_1})$  and the individual components  $J_I$ ,  $J_{II}$ , and  $J_{III}$  along the crack front are shown in Figs 13 and 14, respectively. These results were obtained using the decomposition method. No reference solutions are available for this problem. Solid lines in Figs 13 and 14 are the smooth fit to the domain  $D_2$  solutions.

The  $J$ -values for the three domains agree well with one another. The maximum difference between the results of the domains  $D_2$  and  $D_3$  is about 3%. As expected, (a) all the  $J$  values are symmetric about  $\phi = 90^\circ$ , (b)  $J_I$  and  $J_{II}$  have the highest values at  $\phi = 0$  and  $180^\circ$  with the lowest value at  $\phi = 90^\circ$ , (c) the  $J_{III}$  value is zero at  $\phi = 0$  and  $180^\circ$ , (d) the  $J_{III}$  value is maximum at  $\phi = 90^\circ$  and (e) the  $J_{II}$  value is zero at  $\phi = 90^\circ$ .

*Application to eight-node isoparametric elements*

The use of eight-noded isoparametric elements in the EDI analysis was demonstrated using two classical crack problems in a tension specimen: an embedded elliptic crack and a penny-shaped

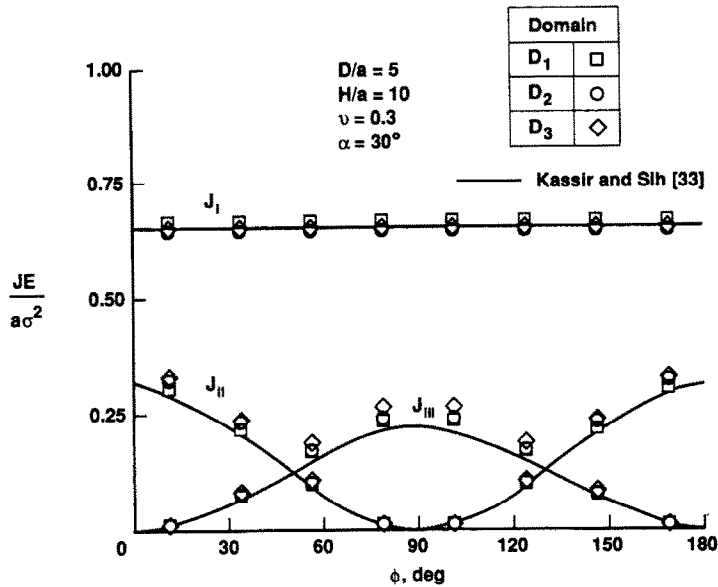


Fig. 12a. Normalized  $J_I$ ,  $J_{II}$ , and  $J_{III}$  distribution along the crack front of an embedded inclined penny-shaped crack.

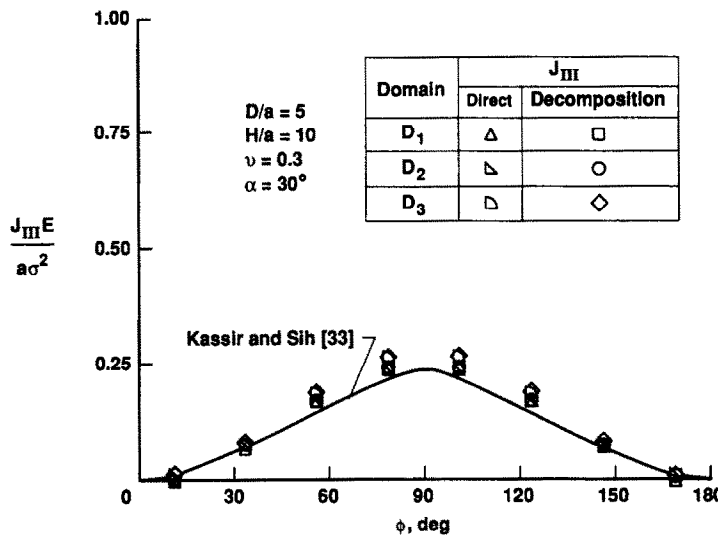


Fig. 12b. Comparison of  $J_{III}$  from direct and decomposition methods.

surface crack. The FE model and the domain used were same as given in ref. [34], except that the notch radius was zero. The FE mesh had 3420 nodes and 2772 eight-noded elements. Around the crack front, a rectangular arrangement of non-singular elements was used (similar to that shown in Fig. 8c). The element size around the crack front was  $a/20$ , where  $a$  is the crack length in the thickness direction of the specimen. A domain consisting of the second ring of elements around the crack front and two element layers along the crack front was used for the  $J$ -integral calculation.

Figure 15 compares the distribution of the normalized total  $J(J_{x_1})$  along the crack front for an embedded elliptic crack calculated from the EDI method to Green and Sneddon's infinite body solution [35]. The two solutions agree well with each other.

Figure 16 shows the distribution of the normalized total  $J$  (which is same as  $J_I$ ) along the crack front for a semi-circular surface crack in a tension specimen. The  $J$ -values from the EDI method agree very well with those from the VCCT [6] and Raju and Newman's solutions [36]. The EDI algorithm is incorporated in the ZIP3D [37] code. The ZIP3D is an elastic and elastic-plastic FE code used to analyze cracked bodies.

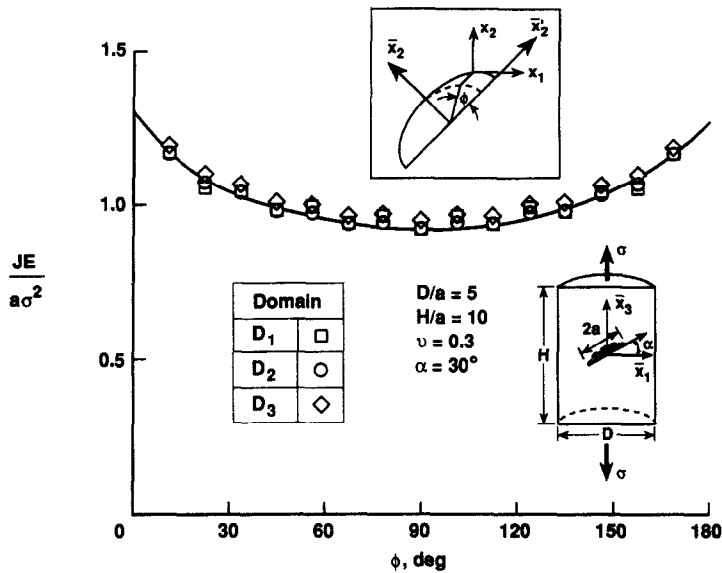


Fig. 13. Normalized total  $J$  distribution along the crack front of an inclined semi-circular surface crack.

CONCLUDING REMARKS

Details of development of the 3-D EDI method for the calculation of mixed-mode fracture mechanics parameters in isotropic or anisotropic and linear or nonlinear materials are presented. Differences and improvements between the current algorithm and that reported in the literature are highlighted. Results presented in this paper are restricted to isotropic elastic solutions. Several single and mixed-mode loaded cracked bodies were analyzed and results were found to agree very well with those available in the literature.

The EDI method with 20- or eight-noded isoparametric, non-singular elements and either a polar or rectilinear arrangement of elements at the crack front gives accurate values of the  $J$ -integral. A simple linear  $S$ -function in the radial direction is recommended if only one ring of elements at the crack front is used. The EDI method was found to be independent of the type of  $S$ -function, except for one special case. The  $J$  values were found to be inaccurate for domains

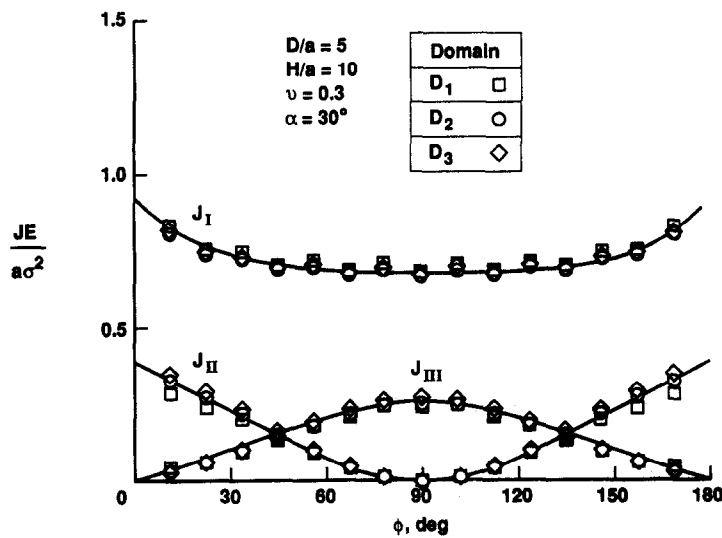


Fig. 14. Normalized  $J_I$ ,  $J_{II}$ , and  $J_{III}$  distribution along the crack front of an inclined semi-circular surface crack.

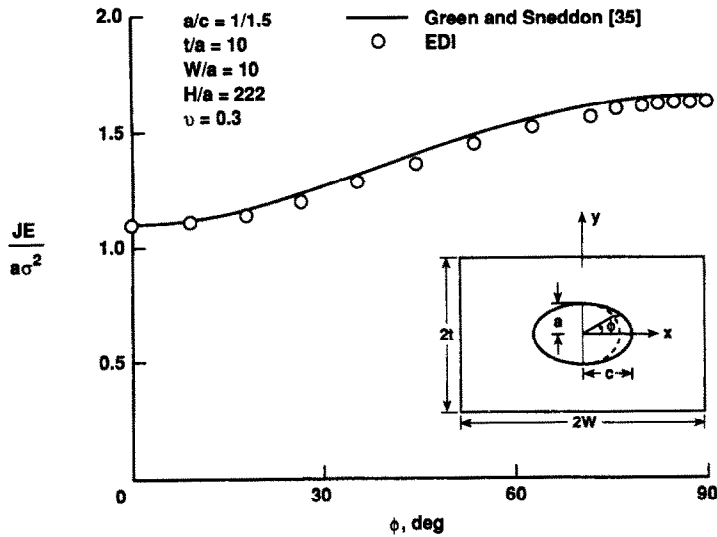


Fig. 15. Application of EDI method to eight-node isoparametric element.

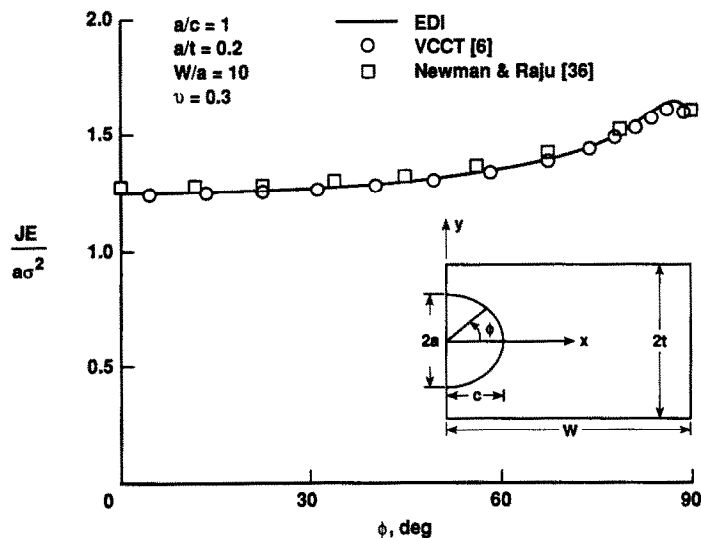


Fig. 16. Comparison of normalized  $J$  distribution for a semi-circular surface crack from EDI, VCCT, and force methods.

consisting of only one ring of elements around the crack front with  $S$ -functions that were quadratic in the radial direction from the crack front.

The EDI method is domain independent provided the radius of the inner surface of the domain is either zero or very small (less than one-tenth of the major crack length). Domains consisting of only the second ring of elements or the first two rings of elements around the crack front reduce the data preparation effort and also give accurate  $J$ -integral values.

The principal advantage of the 3-D EDI method is that the FE idealization need not be orthogonal to the crack front. The orthogonality of the modeling at the crack front is a requirement for the virtual crack closure and the force methods. In the case of mixed-mode loadings, the decomposition method yielded accurate  $J$ -integrals. The method requires the evaluation of only one integral with different sets of displacement and stress fields. However, the method requires an FE mesh that is symmetric about the crack plane.

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## APPENDIX: NUMERICAL IMPLEMENTATION OF THE EDI METHOD

This appendix presents the numerical implementation of the EDI method for an FE analysis with isoparametric elements. The procedure presented in this appendix is similar to that of the 2-D analysis in ref. [23] and is applicable to both eight-noded and 20-noded 3-D isoparametric elements. For the purpose of illustration, the 20-noded element is used.

A typical FE model around the crack front is shown in Fig. A1a. The shaded region represents a typical domain surrounding the crack front. Although no restriction was imposed on the number of elements in the domain either in  $x_1$ - or  $x_2$ -directions, one ring of elements in  $x_1$ - and  $x_2$ -directions was used to explain the procedure. The procedure for computing  $J_{x_k}$  is presented but can be easily extended to the  $J_{III}$  computation as well.

The total  $J$ -integral ( $J_{x_k}$ ) is equivalent to the sum of the domain integrals contributed by the elements in the shaded region shown in Fig. A1.

$$(J_{x_k})_{\text{domain}} = \sum_{i=1}^{N_e} J_{x_{ki}}, \quad (\text{A1})$$

where  $J_{x_{ki}}$  is the volume integral over the  $i$ th element in the shaded region and  $N_e$  is the number of elements enclosed in the domain.

For isoparametric FEs the displacements within the element are defined by the shape functions  $N_j$  and the nodal displacements  $(u_\alpha)_j$ .

$$u_\alpha = N_j(u_\alpha)_j, \quad (\text{A2})$$

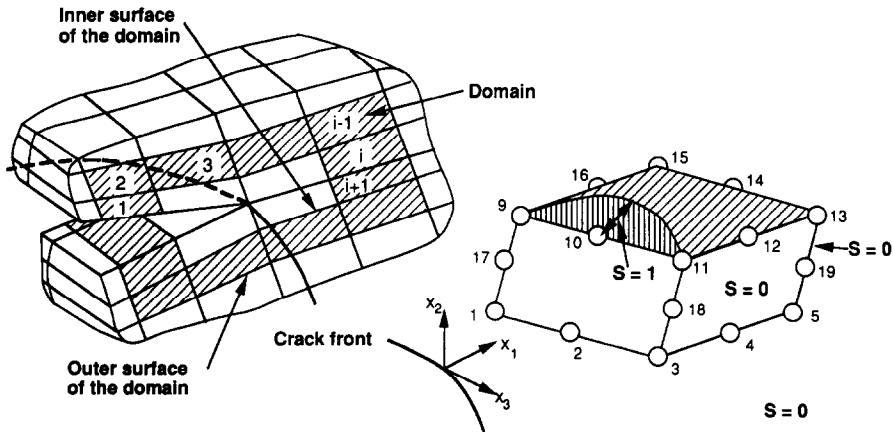
where  $N_j = N_j(\xi, \eta, \zeta)$  and  $\xi, \eta, \zeta$  are the coordinates of the parent element. The index  $j$  takes the value 1– $N_n$  ( $N_n$  is the number of nodes per element;  $N_n = 8$  for an eight-node linear element and  $N_n = 20$  for a 20-node quadratic element) and  $\alpha$  takes the values 1, 2, and 3 corresponding to the displacements in  $x_1$ -,  $x_2$ -, and  $x_3$ -directions, respectively.

The volume integral  $J_{x_{ki}}$  of the  $i$ th element [eq. (20)] is computed using Gaussian quadratures as

$$\begin{aligned} (J_{x_{ki}})_{\text{domain}} \cdot f &= - \left[ \int_{-1}^1 \int_{-1}^1 \int_{-1}^1 \left[ W \frac{\partial S}{\partial x_k} - \{u'_{x_k}\}^T [\underline{\sigma}] \{S'\} \right] (\det[\mathbf{J}]) d\xi d\eta d\zeta \right. \\ &\quad \left. + \int_{-1}^1 \int_{-1}^1 \int_{-1}^1 \left[ \frac{\partial W}{\partial x_k} - \{\sigma\}^T \{\epsilon'_{x_k}\} \right] S (\det[\mathbf{J}]) d\xi d\eta d\zeta \right]_i \\ &= - \left[ \sum_{m=1}^{M_G} \sum_{n=1}^{M_G} \sum_{p=1}^{M_G} \left\{ \left[ W \frac{\partial S}{\partial x_k} - \{u'_{x_k}\}^T [\underline{\sigma}] \{S'\} \right] + \left[ \frac{\partial W}{\partial x_k} - \{\sigma\}^T \{\epsilon'_{x_k}\} \right] S \right\} w_m w_n w_p (\det[\mathbf{J}]) \right]_i, \end{aligned} \quad (\text{A3})$$

where  $M_G$  is the number of Gaussian quadrature points used in each direction  $\xi, \eta$ , and  $\zeta$ ,  $w_m, w_n$ , and  $w_p$  are the Gaussian weights, and  $\det[\mathbf{J}]$  is the determinant of the Jacobian matrix  $[\mathbf{J}]$  defined by

$$[\mathbf{J}] = \begin{bmatrix} \frac{\partial N_1}{\partial \xi} & \frac{\partial N_2}{\partial \xi} & \dots & \frac{\partial N_{N_n}}{\partial \xi} \\ \frac{\partial N_1}{\partial \eta} & \frac{\partial N_2}{\partial \eta} & \dots & \frac{\partial N_{N_n}}{\partial \eta} \\ \frac{\partial N_1}{\partial \zeta} & \frac{\partial N_2}{\partial \zeta} & \dots & \frac{\partial N_{N_n}}{\partial \zeta} \end{bmatrix} \begin{bmatrix} (x_1)_1 & (x_2)_1 & (x_3)_1 \\ (x_1)_2 & (x_2)_2 & (x_3)_2 \\ \vdots & \vdots & \vdots \\ (x_1)_{N_n} & (x_2)_{N_n} & (x_3)_{N_n} \end{bmatrix}. \quad (\text{A4})$$



(a) Domain around the crack front

(b) Typical element  $i$  and distribution of  $S$ -function

Fig. A1. Typical finite-element model and a domain description near the crack front.

Most of the quantities necessary for eqs (20)–(23),  $W$ ,  $W^{III}$ ,  $\{\sigma\}$ ,  $\{\sigma_3\}$ , and  $\{\sigma\}$ , are readily known in terms of the nodal displacements of the element. However, computation of the terms  $S$ ,  $\{S'\}$ , and  $(\partial W)/(\partial x_k)$  needs special attention and is discussed below. Evaluation of the derivative matrices  $\{\epsilon'_{xx}\}$  and  $\{u''_3\}$  is the same as that for  $(\partial W)/(\partial x_k)$ ; hence, they are not discussed.

### S-functions

As mentioned previously, the  $S$ -function is any arbitrary but continuous function with a non-zero value (varying between 0 and 1) on the surface of the inner tube ( $A_i$ ) and a value of zero on the surface of the outer tube and at the two ends of the tube (see Fig. 1). The variation of the  $S$ -function over a typical  $i$ th element in the shaded region is shown in Fig. A1b. The function is conveniently defined using the element shape functions as

$$S(\xi, \eta, \zeta) = N_j S_j, \quad (A5)$$

where  $j = 1$  to  $N_e$  and  $S_j$  is the nodal value of the  $S$ -function at node  $j$  on the element. Different  $S$ -functions can be defined by assigning 0, 0.5 or 1 to  $S_j$ . For a typical element shown in Fig. A1b, the  $S$ -function is completely defined by specifying  $S_2 = S_{10} = 1$  and zero to all other nodes. This definition yields an  $S$ -function having a parabolic variation along the crack front and a linear variation in the radial direction (type I  $S$ -function).

### Partial derivatives of $S$

Once  $S$  is defined the partial derivatives of  $S$ ,  $(\partial S)/(\partial x_1)$ ,  $(\partial S)/(\partial x_2)$ , and  $(\partial S)/(\partial x_3)$ , can be computed using the isoparametric formulation as

$$\begin{Bmatrix} \frac{\partial S}{\partial x_1} \\ \frac{\partial S}{\partial x_2} \\ \frac{\partial S}{\partial x_3} \end{Bmatrix} = [J]^{-1} \begin{Bmatrix} \frac{\partial S}{\partial \xi} \\ \frac{\partial S}{\partial \eta} \\ \frac{\partial S}{\partial \zeta} \end{Bmatrix}, \quad (A6)$$

where  $[J]$  is the Jacobian matrix defined in eq. (A4).

### Partial derivatives of $W$

The terms  $(\partial W)/(\partial x_k)$  are computed by fitting a bilinear equation (in terms of the parent coordinates  $\xi$ ,  $\eta$ , and  $\zeta$ ) to  $W$ , using the values at the  $2 \times 2 \times 2$  integration points and then taking derivatives with respect to  $x_k$ . In ref. [20], the integral  $\int \partial W / \partial x_k dV$  was approximated by evaluating  $(\partial W)/(\partial x_k)$  at the center of the element. A different approach is taken here. Because all the quantities are known at the integration points, the integration is carried out without further approximations of other terms in eqs (20)–(23). The values of the stresses are known to be more reliable at the  $2 \times 2 \times 2$  Gaussian points within the element (in comparison to the nodal values). The stress-work density  $W$  is approximated in bilinear form as

$$W(\xi, \eta, \zeta) = a_1 + a_2 \xi + a_3 \eta + a_4 \zeta + a_5 \xi \eta + a_6 \eta \zeta + a_7 \xi \zeta + a_8 \xi \eta \zeta. \quad (A7)$$

Using the  $2 \times 2 \times 2$  Gaussian values of the stress-work density,  $W$ , eq. (A7) is rewritten as

$$W(\xi, \eta, \zeta) = [1 \ \xi \ \eta \ \zeta \ \xi \eta \ \eta \zeta \ \xi \zeta \ \xi \eta \zeta][T]\{W_G\}, \quad (A8)$$

where

$$[T] = \frac{1}{8} \begin{bmatrix} 1 & 1 & 1 & 1 & 1 & 1 & 1 & 1 \\ -\sqrt{3} & -\sqrt{3} & -\sqrt{3} & -\sqrt{3} & \sqrt{3} & \sqrt{3} & \sqrt{3} & \sqrt{3} \\ -\sqrt{3} & -\sqrt{3} & \sqrt{3} & \sqrt{3} & -\sqrt{3} & -\sqrt{3} & \sqrt{3} & \sqrt{3} \\ -\sqrt{3} & \sqrt{3} & -\sqrt{3} & \sqrt{3} & -\sqrt{3} & \sqrt{3} & -\sqrt{3} & \sqrt{3} \\ 3 & 3 & -3 & -3 & -3 & -3 & 3 & 3 \\ 3 & -3 & -3 & 3 & 3 & -3 & -3 & 3 \\ 3 & -3 & 3 & -3 & -3 & 3 & -3 & 3 \\ -3\sqrt{3} & 3\sqrt{3} & 3\sqrt{3} & -3\sqrt{3} & 3\sqrt{3} & -3\sqrt{3} & -3\sqrt{3} & 3\sqrt{3} \end{bmatrix} \quad (A9)$$

and

$$\{W_G\}^T = \{W_I \ W_{II} \ W_{III} \ W_{IV} \ W_V \ W_{VI} \ W_{VII} \ W_{VIII}\}^T, \quad (A10)$$

where  $W_I$ – $W_{VIII}$  are the values of  $W$  at the  $2 \times 2 \times 2$  Gaussian points shown in Fig. A2. The partial derivatives  $(\partial W)/(\partial \xi)$ ,  $(\partial W)/(\partial \eta)$ , and  $(\partial W)/(\partial \zeta)$  are

$$\begin{Bmatrix} \frac{\partial W}{\partial \xi} \\ \frac{\partial W}{\partial \eta} \\ \frac{\partial W}{\partial \zeta} \end{Bmatrix} = \begin{bmatrix} 0 & 1 & 0 & 0 & \eta & 0 & \zeta & \eta \zeta \\ 0 & 0 & 1 & 0 & \xi & \zeta & 0 & \xi \zeta \\ 0 & 0 & 0 & 1 & 0 & \xi & \eta & \xi \eta \end{bmatrix} [T][W_G]. \quad (A11)$$



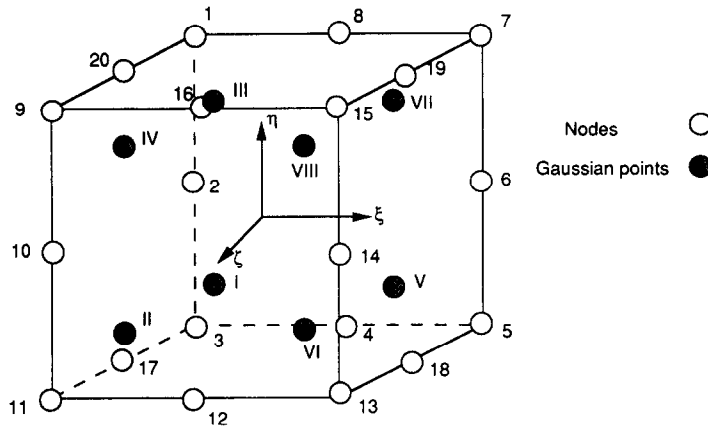


Fig. A2. Nodes and Gaussian quadrature points ( $2 \times 2 \times 2$ ) in a 20-node isoparametric element.

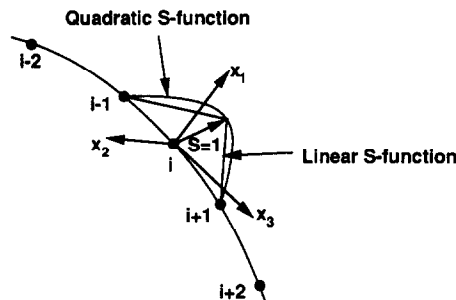


Fig. A3. Discretization of the crack front and linear and quadratic  $S$ -function.

The derivatives  $(\partial W)/(\partial x_k)$  can now be obtained as

$$\begin{Bmatrix} \frac{\partial W}{\partial x_1} \\ \frac{\partial W}{\partial x_2} \\ \frac{\partial W}{\partial x_3} \end{Bmatrix} = [J]^{-1} \begin{Bmatrix} \frac{\partial W}{\partial \xi} \\ \frac{\partial W}{\partial \eta} \\ \frac{\partial W}{\partial \zeta} \end{Bmatrix}, \quad (\text{A12})$$

where  $[J]$  is defined in eq. (A4).

Similarly, derivatives of  $\{\epsilon_{xk}\}$  and  $\{u_3^s\}$  can be obtained. All the necessary terms in eqs (20)–(24) are now known and, hence, the domain integrals for each element can be calculated.

#### Computation of $J$ -integral along the crack front

In a 3-D FE model of a cracked body, the crack front is divided into a number of segments. To calculate the  $J$ -integral at each of the nodal points, for example at node  $i$ , consider the crack front segment between the nodes  $(i - 1)$  and  $(i + 1)$  (see Fig. A3). The  $S$ -function will have a value of unity at node  $i$  and zero at nodes  $(i - 1)$  and  $(i + 1)$ . Since the  $S$ -function is generated from the element shape functions, the  $S$ -function is linear for the eight-noded linear element and quadratic for the 20-noded quadratic element (see Fig. A3). Utilizing the domain corresponding to the crack front segment  $(i - 1)$  and  $(i + 1)$  (see, for example Fig. A1 for a 20-node model), the  $J$ -integrals are calculated from eqs (15) and (16). The analysis is repeated at other nodal locations. If the first node is on the plane of symmetry of the model, the left-half [segment  $(i - 1)$  to  $i$ ] of the  $S$ -function is neglected. However, the accuracy of the  $J$ -integral is poor at nodes on the free surface of the model because of the well known boundary layer effect [38, 39].

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