

Write our kinematic constraints and symbolic variables

```
syms F tau M m L t g x(t) theta(t)

% Cart Position is just x
disp_eq(x, x, theta, t, "Cart Position");
```

Cart Position:

x

```
% Pendulum Position is function of x and theta
x_p = x + L * sin(theta);
y_p = L * cos(theta); % theta is deviation from vertical
disp_eq(x_p, x, theta, t, "Pendulum x Position");
```

Pendulum x Position:

$x + L \sin(\theta)$

```
disp_eq(y_p, x, theta, t, "Pendulum y Position");
```

Pendulum y Position:

$L \cos(\theta)$

```
% Cart Velocity
v_x = diff(x, t);
disp_eq(v_x, x, theta, t, "Cart Velocity");
```

Cart Velocity:

\dot{x}

```
% Pendulum Velocity
v_px = diff(x_p, t);
v_py = diff(y_p, t);
disp_eq(v_px, x, theta, t, "Pendulum x Velocity");
```

Pendulum x Velocity:

$\dot{x} + L \dot{\theta} \cos(\theta)$

```
disp_eq(v_py, x, theta, t, "Pendulum y Velocity");
```

Pendulum y Velocity:

$-L \dot{\theta} \sin(\theta)$

Find formulas for energies

```
% Cart Kinetic Energy T
T_c = 1/2 * M * diff(x, t) ^ 2;
disp_eq(T_c, x, theta, t, "Cart KE");
```

Cart KE:

$$\frac{M \dot{x}^2}{2}$$

% Pendulum Kinetic Energy T

```
T_p = simplify(1/2 * m * (v_px^2 + v_py^2));  
disp_eq(T_p, x, theta, t, "Pendulum KE");
```

Pendulum KE:

$$\frac{m (L^2 \dot{\theta}^2 + 2 \cos(\theta) L \dot{\theta} \dot{x} + \dot{x}^2)}{2}$$

% Total KE

```
T = simplify(T_c + T_p);  
disp_eq(T, x, theta, t, "Total KE");
```

Total KE:

$$\frac{m (L^2 \dot{\theta}^2 + 2 \cos(\theta) L \dot{\theta} \dot{x} + \dot{x}^2)}{2} + \frac{M \dot{x}^2}{2}$$

% Potential energy of cart is zero

```
disp_eq(0, x, theta, t, "Cart PE");
```

Cart PE:

$$0$$

% Pendulum PE = Total PE:

```
V = m * g * L * cos(theta);  
disp_eq(V, x, theta, t, "Pendulum PE");
```

Pendulum PE:

$$L g m \cos(\theta)$$

Calculate the lagrangian and the lagrangian equations for our system

% Lagrangian

```
lagrange = simplify(T - V);  
L_sym = disp_eq(lagrange, x, theta, t, "Lagrangian");
```

Lagrangian:

$$\frac{M \dot{x}^2}{2} + \frac{m \dot{x}^2}{2} + \frac{L^2 m \dot{\theta}^2}{2} - L g m \cos(\theta) + L m \dot{\theta} \dot{x} \cos(\theta)$$

% External Force associated with x:

$$Q_x = F;$$

```
% External Torque associated with theta:
Q_theta = 0;
```

```
% Apply lagrange's equation
```

```
eq_1 = diff(diff(lagrange, diff(x, t)), t) - diff(L, x) == Q_x;
eq_2 = diff(diff(lagrange, diff(theta, t)), t) - diff(L, theta) == Q_theta;
eq_1_sym = disp_eq(eq_1, x, theta, t, "Lagrange Equation 1");
```

Lagrange Equation 1:

$$-L m \sin(\theta) \ddot{\theta}^2 + M \ddot{x} + m \ddot{x} + L m \ddot{\theta} \cos(\theta) = F$$

```
eq_2_sym = disp_eq(eq_2, x, theta, t, "Lagrange Equation 2");
```

Lagrange Equation 2:

$$L^2 m \ddot{\theta} + L m \ddot{x} \cos(\theta) - L m \dot{\theta} \dot{x} \sin(\theta) = 0$$

Solve for: \ddot{x} and $\ddot{\theta}$

```
syms x_dot theta_dot x_ddot theta_ddot
```

```
[x_ddot, theta_ddot] = solve([eq_1_sym, eq_2_sym], [x_ddot, theta_ddot]);
x_ddot = simplify(x_ddot);
theta_ddot = simplify(theta_ddot);
disp("Solutions for x_ddot and theta_ddot:")
```

Solutions for x_ddot and theta_ddot:

```
disp(x_ddot)
```

$$\frac{L m \sin(\theta) \ddot{\theta}^2 - m \dot{x} \cos(\theta) \sin(\theta) \dot{\theta} + F}{-m \cos(\theta)^2 + M + m}$$

```
disp(theta_ddot)
```

$$-\frac{F \cos(\theta) - M \dot{\theta} \dot{x} \sin(\theta) - m \dot{\theta} \dot{x} \sin(\theta) + L m \ddot{\theta}^2 \cos(\theta) \sin(\theta)}{L (-m \cos(\theta)^2 + M + m)}$$

```
x_sym = sym('x');
theta_sym = sym('theta');
X = [x_sym, x_dot, theta_sym, theta_dot]; % can't use the function sym
U = [F, tau]; % no torque control so jacobian second col is 0
X_dot = [x_dot, x_ddot, theta_dot, theta_ddot];

% this part is inspired by Jason's matlab code to collect coefficients of X
% in X_dot
```

```
X_dot_x = simplify(collect(X_dot, X));
Jacobi_A = jacobian(X_dot_x, X);
X_dot_u = simplify(collect(X_dot, U));
Jacobi_B = jacobian(X_dot, U);
```

```
disp("Jacobian Matrices A and B:")
```

Jacobian Matrices A and B:

```
disp(Jacobi_A)
```

$$\begin{pmatrix} 0 & 1 & 0 & 0 \\ 0 & -\frac{m \dot{\theta} \cos(\theta) \sin(\theta)}{\sigma_1} & \frac{L m \dot{\theta}^2 \cos(\theta) - m \dot{x} \dot{\theta} \cos(\theta)^2 + m \dot{x} \dot{\theta} \sin(\theta)^2}{\sigma_1} & -\frac{2 m}{\sigma_1} \\ 0 & 0 & 0 & 0 \\ 0 & \frac{M \dot{\theta} \sin(\theta) + m \dot{\theta} \sin(\theta)}{L \sigma_1} & \frac{F \sin(\theta) - L m \dot{\theta}^2 \cos(\theta)^2 + L m \dot{\theta}^2 \sin(\theta)^2 + M \dot{\theta} \dot{x} \cos(\theta) + m \dot{\theta} \dot{x} \cos(\theta)}{L \sigma_1} & + \frac{2}{L} \end{pmatrix}$$

where

$$\sigma_1 = -m \cos(\theta)^2 + M + m$$

```
disp(Jacobi_B)
```

$$\begin{pmatrix} 0 & 0 \\ \frac{1}{-m \cos(\theta)^2 + M + m} & 0 \\ 0 & 0 \\ -\frac{\cos(\theta)}{L (-m \cos(\theta)^2 + M + m)} & 0 \end{pmatrix}$$

Test controllability conditions and see LQR matrices

```
% say cart mass is 5kg, pendulum mass is 0.5kg, L = 1 m, g = 9.81 m/s^2
A = subs(Jacobi_A, [M, m, L, g, F], [5, 0.5, 1, 9.81, 0]);
B = subs(Jacobi_B, [M, m, L, g], [5, 0.5, 1, 9.81]);
fprintf("A and B Matrices: \n")
```

A and B Matrices:

```
disp(A)
```

$$\begin{pmatrix} 0 & 1 & 0 \\ 0 & \frac{\sigma_2}{2\sigma_1} & -\frac{\frac{\dot{\theta}^2 \cos(\theta)}{2} - \frac{\dot{x} \dot{\theta} \cos(\theta)^2}{2} + \frac{\dot{x} \dot{\theta} \sin(\theta)^2}{2}}{\sigma_1} - \frac{\cos(\theta) \sin(\theta) \left(\frac{\dot{\theta}^2 \sin(\theta)}{2} - \frac{\dot{\theta} \dot{x} \cos(\theta) \sin(\theta)}{2} \right)}{\sigma_1^2} \\ 0 & 0 & 0 \\ 0 & -\frac{11 \dot{\theta} \sin(\theta)}{2\sigma_1} - \frac{-\frac{\dot{\theta}^2 \cos(\theta)^2}{2} + \frac{\dot{\theta}^2 \sin(\theta)^2}{2} + \frac{11 \dot{x} \dot{\theta} \cos(\theta)}{2}}{\sigma_1} - \frac{\cos(\theta) \sin(\theta) \left(\frac{11 \dot{\theta} \dot{x} \sin(\theta)}{2} - \frac{\dot{\theta}^2 \cos(\theta) \sin(\theta)}{2} \right)}{\sigma_1^2} \end{pmatrix}$$

where

$$\sigma_1 = \frac{\cos(\theta)^2}{2} - \frac{11}{2}$$

$$\sigma_2 = \dot{\theta} \cos(\theta) \sin(\theta)$$

disp(B)

$$\begin{pmatrix} 0 & 0 \\ -\frac{1}{\frac{\cos(\theta)^2}{2} - \frac{11}{2}} & 0 \\ 0 & 0 \\ \frac{\cos(\theta)}{\frac{\cos(\theta)^2}{2} - \frac{11}{2}} & 0 \end{pmatrix}$$

```
% system is not full rank if jacobian is evaluated at unstable equilibrium point
% as an experiment, I evaluate it with slight angular velocity and
% theta=pi/4
theta_0 = pi/4;
theta_dot_0 = 0.001
```

```
theta_dot_0 = 1.0000e-03
```

```
A_0 = double(subs(A, [x_sym, x_dot, theta_sym, theta_dot], [0, 0, theta_0,
theta_dot_0]))
```

```
A_0 = 4x4
    0    1.0000         0         0
    0   -0.0000    0.0000    0.0001
    0         0         0    1.0000
    0    0.0007    0.0000   -0.0001
```

```
B_0 = double(subs(B, [x_sym, x_dot, theta_sym, theta_dot], [0, 0, theta_0,
theta_dot_0]))
```

```

B_0 = 4x2
      0      0
    0.1905    0
      0      0
   -0.1347    0

```

```
rank(ctrb(A_0,B_0))
```

```
ans = 4
```

```

% cost matrices
Q = diag([10, 0, 10, 0]);
R = diag([0.1, 0.1]);

% lqr gain
[K, S, e] = lqr(A_0, B_0, Q, R);
disp(K)

```

```

1.0e+04 *
   -0.0010    3.2240    0.0014    4.5578
    0.0000   -0.0000   -0.0000   -0.0000

```

```

function equation_s = disp_eq(equation, x, theta, t, name)
    % replace second derivatives
    equation_s = subs(equation, diff(x, t, 2), sym('x_ddot'));
    equation_s = subs(equation_s, diff(theta, t, 2), sym('theta_ddot'));

    % replace first derivatives
    equation_s = subs(equation_s, diff(x, t), sym('x_dot'));
    equation_s = subs(equation_s, diff(theta, t), sym('theta_dot'));

    % replace function terms
    equation_s = subs(equation_s, x, sym('x'));
    equation_s = subs(equation_s, theta, sym('theta'));

    % display equation
    fprintf(name + ":\n")
    disp(equation_s)
    fprintf("\n")
end

```