



Control of a Rotary Inverted Pendulum

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Abstract—This work deals with the control of a rotary inverted pendulum (see Figure 1). This device is composed of the following: an arm rotating in the horizontal plane where one of its ends is mounted on a motor shaft and where a rod is mounted on its other end. The rod's lower end is mounted on the arm's free end in such a manner that, the rod is moving as an inverted pendulum in a plane that is at all times perpendicular to the rotating arm. The problem dealt with here is to find a control law to the motor's output torque such that the inverted pendulum motion will be stabilized about a vertical axis. © 1998 Elsevier Science Ltd. All rights reserved.

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1. INTRODUCTION

This work deals with the control of a rotary inverted pendulum (see Figure 1). This device is composed of the following: an arm rotating in the horizontal plane where one of its ends is mounted on a motor shaft and where a rod is mounted on its other end. The rod's lower end is mounted on the arm's free end in such a manner that, the rod is moving as an inverted pendulum in a plane that is at all times perpendicular to the rotating arm. The problem dealt with here is to find a control law to the motor's output torque such that the inverted pendulum motion will be stabilized about a vertical axis. Such a system is promoted for educational purpose by [1]. For the classical problem of the inverted pendulum see [2].

2. DYNAMICAL MODEL

In this work, we consider the control of the motion of a rotary inverted pendulum. Let \mathbf{I} , \mathbf{J} , and \mathbf{K} be unit vectors along an interial (X, Y, Z) -coordinate system.

Denote by \mathbf{i}_1 a unit vector fixed along the rotating arm, and by \mathbf{j}_1 a unit vector perpendicular to \mathbf{i}_1

$$\mathbf{i}_1 = \cos \alpha \mathbf{I} + \sin \alpha \mathbf{J}, \quad \mathbf{j}_1 = -\sin \alpha \mathbf{I} + \cos \alpha \mathbf{J}. \quad (1)$$

Also, let \mathbf{k} denote a unit vector fixed along the inverted pendulum, and let \mathbf{j} be a unit vector perpendicular to \mathbf{k}

$$\mathbf{k} = \cos \theta \mathbf{j}_1 + \sin \theta \mathbf{K}, \quad \mathbf{j} = -\sin \theta \mathbf{j}_1 + \cos \theta \mathbf{K}. \quad (2)$$

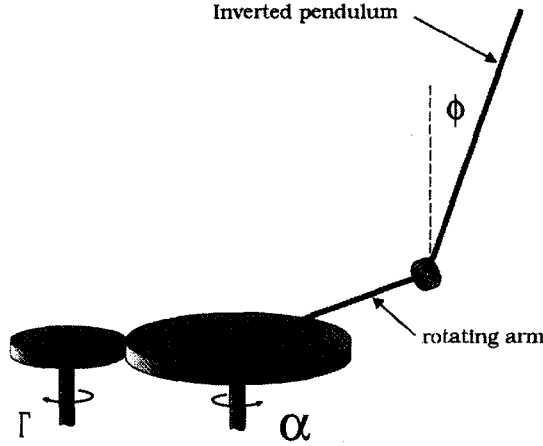


Figure 1. View of the rotary inverted pendulum. Here $\phi = \pi/2 - \theta$.

Thus, it can be shown that the angular velocity vector of the inverted pendulum is given by

$$\omega_P = \frac{d\theta}{dt} \mathbf{i}_1 + \frac{d\alpha}{dt} \sin \theta \mathbf{k} + \frac{d\alpha}{dt} \cos \theta \mathbf{j}. \quad (3)$$

Denote by L_1 the length of the rotating arm and by L_2 the length of the inverted pendulum. Then, the Lagrangian function [3], for the system is given by

$$L = K_R + K_P - V, \quad (4)$$

where

$$K_R = \frac{1}{2} (m_R L_{o1}^2 + I_{R1}) \left(\frac{d\alpha}{dt} \right)^2 \quad (5)$$

is the kinetic energy of the rotating arm,

$$\begin{aligned} K_P = & \frac{1}{2} m_P L_1^2 \left(\frac{d\alpha}{dt} \right)^2 + \frac{1}{2} (m_P L_{o2}^2 + I_{P1}) \left[\left(\frac{d\theta}{dt} \right)^2 + \left(\frac{d\alpha}{dt} \right)^2 \cos^2 \theta \right] \\ & - m_P L_1 L_{o2} \left(\frac{d\alpha}{dt} \right) \left(\frac{d\theta}{dt} \right) \sin \theta \end{aligned} \quad (6)$$

is the kinetic energy of the inverted pendulum, and

$$V = m_P g L_{o2} \sin \theta \quad (7)$$

is the potential energy of the inverted pendulum.

In equations (5)–(7), m_R denotes the mass of the rotating arm, m_P denotes the mass of the inverted pendulum, $L_{o1} = L_1/2$, $L_{o2} = L_2/2$, I_{R1} denotes the moment of inertia of the rotating arm about a vertical axis through its center of mass, $I_{P1} = I_{P3}$ denote moments of inertia of the inverted pendulum about \mathbf{i}_1 and \mathbf{j} (passing through its center of mass). Note that the “slender rod” approximation has been applied to both the rotating arm and the inverted pendulum. Define the following vectors:

$$\mathbf{q} = (\alpha, \theta)^\top, \quad \mathbf{p} = \left(\frac{d\alpha}{dt}, \frac{d\theta}{dt} \right)^\top.$$

Thus, the Lagrange equations for system [3], turn out here to be

$$\frac{d}{dt} \left(\frac{\partial L}{\partial p_1} \right) - \frac{\partial L}{\partial q_1} = \Gamma, \quad \frac{d}{dt} \left(\frac{\partial L}{\partial p_2} \right) - \frac{\partial L}{\partial q_2} = 0, \quad (8)$$

where Γ denotes the torque exerted by the motor on the rotating arm. Denote $\mathbf{\Gamma} = (\Gamma, 0)^\top$. Then, by using equation (8) the following equation is obtained:

$$\mathbf{M}(\mathbf{q}) \frac{d^2 \mathbf{q}}{dt^2} + \mathbf{h}(\mathbf{q}, \mathbf{p}) = \mathbf{\Gamma}, \quad (9)$$

where denoting the components of $\mathbf{M}(\mathbf{q})$ by m_{ij} , $i, j = 1, 2$,

$$m_{11} = I_{O1} + I_{O2} \cos^2 \theta, \quad m_{12} = -I_{12} \sin \theta, \quad m_{21} = m_{12}, \quad m_{22} = I_{O2},$$

where

$$I_{O1} = m_R L_{o1}^2 + I_{R1} + m_P L_1^2, \quad I_{O2} = I_{P1} + m_P L_{o2}^2, \quad I_{12} = m_P L_1 L_{o2}.$$

Also,

$$h_1(\mathbf{q}, \mathbf{p}) = -2 I_{O2} \frac{d\alpha}{dt} \frac{d\theta}{dt} \sin \theta \cos \theta - I_{12} \left(\frac{d\theta}{dt} \right)^2 \cos \theta, \quad (10)$$

$$h_2(\mathbf{q}, \mathbf{p}) = I_{O2} \left(\frac{d\alpha}{dt} \right)^2 \sin \theta \cos \theta + m_P g L_{o2} \cos \theta, \quad (11)$$

In addition,

$$\det \mathbf{M}(\mathbf{q}) = I_{o1} I_{o2} + (I_{o2}^2 + I_{12}^2) \cos^2 \theta - I_{12}^2 > 0. \quad (12)$$

Hence, equation (9) leads to

$$\frac{d^2 \alpha}{dt^2} = F_1 + \frac{m_{22}}{D} \Gamma, \quad \frac{d^2 \theta}{dt^2} = F_2 - \frac{m_{12}}{D} \Gamma, \quad (13)$$

where

$$F_1 = D^{-1}(-h_1 m_{22} + h_2 m_{12}), \quad F_2 = D^{-1}(-h_2 m_{11} + h_1 m_{12}), \quad D = \det \mathbf{M}(\mathbf{q}).$$

Thus, equation (13) constitute the equations of motion for the problem dealt with here. Define the following transformation:

$$v = F_2 - \frac{m_{12}}{D} \Gamma, \quad (14)$$

whose inverse transformation is given by

$$\Gamma = \frac{D}{m_{12}} (F_2 - v), \quad (15)$$

equation (15) is defined for all $0 < \theta < \pi$. Thus, the second equation in equation (13) yields

$$\frac{d^2 \theta}{dt^2} = v. \quad (16)$$

By choosing $v = -k_1 \frac{d\theta}{dt} - k_2(\theta - \pi/2)$ where $k_1 > 0$, $k_2 > 0$, $k_1^2 < 4k_2$, equation (16) yield

$$\frac{d^2 \theta}{dt^2} + k_1 \frac{d\theta}{dt} + k_2 \left(\theta - \frac{\pi}{2} \right) = 0, \quad (17)$$

for all $0 < \theta(0) < \pi$. From equations (17) and (13), it follows that

$$\lim_{t \rightarrow \infty} \theta(t) = \frac{\pi}{2}, \quad \lim_{t \rightarrow \infty} \frac{d\theta(t)}{dt} = 0, \quad (18)$$

and

$$\lim_{t \rightarrow \infty} \frac{d^2 \alpha(t)}{dt^2} = 0. \quad (19)$$

Hence, the control law Γ given by (15), where

$$v = -k_1 \frac{d\theta}{dt} - k_2 \left(\theta - \frac{\pi}{2} \right) \quad (20)$$

is the required control law.

REMARK. Suppose that

$$\mathbf{k} = \cos \theta \mathbf{i}_1 + \sin \theta \mathbf{K}, \quad (21)$$

that is, the motion of the inverted pendulum is confined to the $(\mathbf{i}_1, \mathbf{K})$ plane. Then, it can be shown, by using the methods of this paper, that the matrix $\mathbf{M}(\mathbf{q})$ (equation (9)), is diagonal. That is, the motion of the inverted pendulum will be completely uncontrollable.

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