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Classical Mechanics

CH. 6 OSCILLATIONS LECTURE NOTES

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Oscillations & Normal Modes (ch. 6)

Generic Problem:

* System's w/ $n \gg 1$ degrees of freedom

* Complicated potential that is a function of degrees of freedom

$$\{q_i\} \longrightarrow L(\{q_i\}, \{\dot{q}_i\}) = T - V$$

Question: If $V(\{q_i\})$ is "complicated" are there simple regimes we can solve/treat?

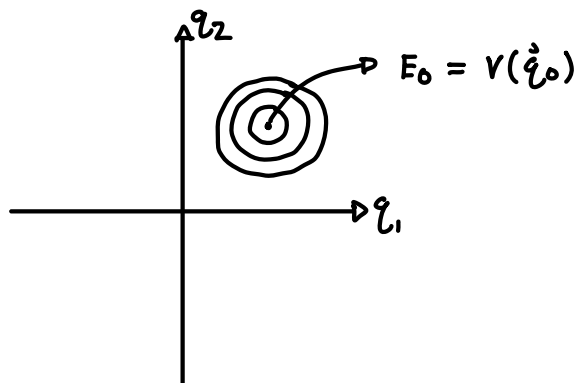
Answer: Possible near equilibrium

Equilibrium: Generalized Forces Vanish

$$Q_j = \left(-\frac{\partial V}{\partial q_j} \right) \bigg|_{\dot{q} = \dot{q}_0} = 0$$

Could be: * minimum \longrightarrow stable

* maximum \longrightarrow unstable



\Rightarrow Want to approximate Lagrangian near equilibrium

\Rightarrow Define new co-ordinates $q_j = \bar{q}_j + \eta_j$ \leftarrow small or displaced co-ordinates $\bar{q}_j = q_j - q_{j,0}$

Next, Taylor expand potential about equilibrium:

$$V(\bar{q}) \approx V(\bar{q}_0) + \sum_j \left. \frac{\partial V}{\partial q_j} \right|_{\bar{q} = \bar{q}_0} \eta_j + \frac{1}{2} \sum_{j,k} \left. \frac{\partial^2 V}{\partial q_j \partial q_k} \right|_{\bar{q} = \bar{q}_0} \eta_j \eta_k + O(\eta^3)$$

Note:

① $\left. \frac{\partial V}{\partial q_j} \right|_{\bar{q} = \bar{q}_0} = 0 \longrightarrow$ Terms $O(\eta_1)$ in V vanish

② η_j is small so that we truncate the expansion after $O(\eta^2)$

$$V(\vec{q}) = \frac{1}{2} \sum_{jk} V_{jk} q_j q_k \quad \rightarrow \quad \left. \frac{\partial^2 V}{\partial q_j \partial q_k} \right|_{\vec{q}=\vec{q}_0} = V_{jk} = V_{kj}$$

\Rightarrow Similar for kinetic energy $T = \frac{1}{2} \sum_{jk} m_{jk} \dot{q}_j \dot{q}_k$ (homogeneous quadratic function of \dot{q}), $T = \frac{1}{2} \sum_{jk} T_{jk} \dot{q}_j \dot{q}_k \rightarrow$ Somehow depends on m_{jk}

Together, approximate Lagrangian, $L \approx \frac{1}{2} \sum_{jk} T_{jk} \dot{q}_j \dot{q}_k - \frac{1}{2} \sum_{jk} V_{jk} q_j q_k$
 \rightarrow quadratic in q & \dot{q}

Lagrangian \rightarrow E.O.M: $\left\{ \sum_k T_{jk} \dot{q}_k + V_{jk} q_k = 0 \right\} (*)$ (n) coupled EOM for n d.o.F

Eigenvalue Equation (b.2)

\Rightarrow Have system of coupled oscillators, Guess: $q_j = a_j e^{-i\omega t}$ \rightarrow Angular frequency of oscillation
 \rightarrow Amplitude of oscillation

$\ddot{q}_j = -\omega^2 q_j \Rightarrow$ Plug into (*)

$$\rightarrow \left\{ \sum_k -\omega^2 T_{jk} q_k + V_{jk} q_k = 0 \right\} \rightarrow \left\{ \sum_k -\omega^2 T_{jk} a_k + V_{jk} a_k = 0 \right\}$$

$$(-\omega^2 I + V) \vec{a} = 0, \quad \vec{a} = \begin{pmatrix} a_1 \\ a_2 \\ \vdots \\ a_n \end{pmatrix} : \text{Immediate solution, } \vec{a} = \vec{0} \rightarrow \text{no motion, boring}$$

Look for case where determinant of $(-\omega^2 I + V)$ Vanishes

i) Solution of vanishing det supplies ω

ii) Obtain $\{a_k\}$ by plugging back into original equation

Generalized eigenvalues problem, $VA = \lambda TA$ $\begin{pmatrix} \lambda_1 & & \\ & \lambda_2 & \\ & & \lambda_3 \end{pmatrix}$ Eigenvalues related to solution of ω
 $A \equiv (\vec{a}^{(1)}, \vec{a}^{(2)}, \vec{a}^{(3)})$

Eigenvalues from,

$$\det \begin{bmatrix} V_{11} - \omega^2 T_{11} & V_{12} - \omega^2 T_{12} & \dots \\ V_{21} - \omega^2 T_{21} & \dots & \dots \\ \vdots & \vdots & \vdots \end{bmatrix}$$

Keep in mind $V_{jk} = V_{kj}$ & $T_{jk} = T_{kj}$ & real

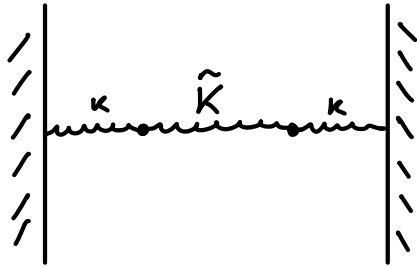
$q_j = a_j e^{-i\omega t} \rightarrow$ General solution: $q_j = \sum_{\alpha} C_{\alpha} a_{j\alpha} e^{-i\omega_{\alpha} t}$
 \rightarrow j th component
 \rightarrow Sum over solution
 \rightarrow Sum over oscillators w/ frequency ω_{α}

$\eta_j = \sum_{\alpha} a_{j\alpha} \xi_{\alpha} \rightarrow$ uncoupled oscillator co-ordinates \Rightarrow Normal mode co-ordinates

or, $\dot{\vec{\eta}} = A \dot{\vec{\xi}}$, Returning to potential: $V \equiv \frac{1}{2} \dot{\vec{\eta}}^T V \dot{\vec{\eta}} = \frac{1}{2} \dot{\vec{\xi}}^T \underbrace{A^T V A}_{\text{relative to } \omega_{\alpha}^2} \dot{\vec{\xi}}$

$V = \frac{1}{2} \sum_{\alpha} \omega_{\alpha}^2 \xi_{\alpha}^2$, $\omega_{\alpha} \Rightarrow$ normal mode oscillators

Worked Example: Pair of masses coupled by three springs to walls.



Define: $\eta_{1,2} \Rightarrow$ Displacement of masses from equilibrium position

Lagrangian: $L = \frac{1}{2} m \dot{\eta}_1^2 + \frac{1}{2} m \dot{\eta}_2^2 - \frac{k}{2} \eta_1^2 - \frac{k}{2} \eta_2^2 - \frac{\tilde{k}}{2} (\eta_1 - \eta_2)^2$

$T = \begin{pmatrix} m & 0 \\ 0 & m \end{pmatrix}$, $V = - \begin{pmatrix} k + \tilde{k} & \partial k \\ \partial k & k + \tilde{k} \end{pmatrix}$

11-1-21

Normal Mode Analysis: Complex model w/ many degrees of freedom \rightarrow Intractable \Rightarrow Simplify? Near equilibrium effective quadratic lagrangian.

$\left. \frac{\partial V}{\partial q_j} \right|_{q_{0j}} = 0$, $\eta_j = q_j - q_{0j}$ displaced co-ordinates

i) $L \approx \frac{1}{2} \sum T_{jk} \dot{\eta}_j \dot{\eta}_k - V_{jk} \eta_j \eta_k \longrightarrow$ system of coupled oscillators

ii) Generate E.O.M $\{ \sum_k T_{jk} \ddot{\eta}_k + V_{jk} \eta_k = 0 \}$

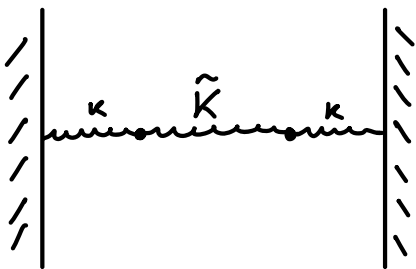
iii) Guess: $\eta_j = a_j e^{-i\omega t} \rightarrow$ Matrix eqn: $(-\omega^2 I + V) \vec{a} = 0$

\Rightarrow Generalized evalue problem

iv) Solve for evalues: $\lambda \sim \omega^2$, evector \rightarrow supplied a_j

v) $\eta_j = \sum_k c_k \vec{a}^{(k)} e^{-i\omega_k t} \rightarrow \vec{\eta} = A \vec{\xi} \rightarrow (\vec{a}^{(1)}, \vec{a}^{(2)}, \dots)$ $L \rightarrow \sum_k \frac{1}{2} \dot{\xi}_k^2 + \omega_k^2 / 2 \xi_k^2$

Worked Example



$\eta_{1,2}$: displacement of masses from equilibrium configuration

Lagrangian: $L = \frac{m}{2} \dot{\eta}_1^2 + \frac{m}{2} \dot{\eta}_2^2 - \frac{k}{2} \eta_1^2 - \frac{k}{2} \eta_2^2$, $T = \begin{pmatrix} m & 0 \\ 0 & m \end{pmatrix}$, $V = \begin{pmatrix} k + \tilde{k} & -k \\ -k & k + \tilde{k} \end{pmatrix}$

EOM

$m \ddot{\eta}_1 = -k \eta_1 - \tilde{k} (\eta_1 - \eta_2)$, $m \ddot{\eta}_2 = -k \eta_2 - \tilde{k} (\eta_2 - \eta_1)$, guess: $\eta_j = a_j e^{-i\omega t}$

$\ddot{\eta}_j = -\omega^2 \eta_j = -\omega^2 a_j e^{-i\omega t}$, generate matrix equation:

$$\begin{pmatrix} -m\omega^2 + k + \tilde{k} & -\tilde{k} \\ -\tilde{k} & -m\omega^2 + k + \tilde{k} \end{pmatrix} \begin{pmatrix} a_1 \\ a_2 \end{pmatrix} = 0, \det(\dots) = 0 \rightarrow (-m\omega^2 + k + \tilde{k})^2 - \tilde{k}^2 = 0$$

$$\omega^2 = k/m \rightsquigarrow \lambda_1 \text{ or } (k + 2\tilde{k})/m \rightsquigarrow \lambda_2, \omega = \pm \sqrt{\frac{k}{m}} \text{ or } \omega = \pm \sqrt{\frac{k+2\tilde{k}}{m}}$$

For $\omega^2 = k/m$:

$$k \begin{pmatrix} 1 & -1 \\ -1 & 1 \end{pmatrix} \begin{pmatrix} a_1 \\ a_2 \end{pmatrix} = \begin{pmatrix} 0 \\ 0 \end{pmatrix}, \text{eigenvector: } \propto \begin{pmatrix} 1 \\ 1 \end{pmatrix} \rightsquigarrow a_1 - a_2 = 0 \rightarrow a_1 = a_2$$

For $\omega^2 = (2k + \tilde{k})/m$

$$k \begin{pmatrix} -1 & -1 \\ -1 & -1 \end{pmatrix} \begin{pmatrix} a_1 \\ a_2 \end{pmatrix} = \begin{pmatrix} 0 \\ 0 \end{pmatrix} \Rightarrow a_1 + a_2 = 0 \therefore a_1 = -a_2, \text{eigenvector: } \propto \begin{pmatrix} 1 \\ -1 \end{pmatrix}$$

General Solution,

$$\begin{pmatrix} \eta_1 \\ \eta_2 \end{pmatrix} = c_1 \begin{pmatrix} 1 \\ 1 \end{pmatrix} e^{i\omega_S t} + c_2 \begin{pmatrix} 1 \\ 1 \end{pmatrix} e^{-i\omega_S t} + c_3 \begin{pmatrix} 1 \\ -1 \end{pmatrix} e^{i\omega_F t} + c_4 \begin{pmatrix} 1 \\ -1 \end{pmatrix} e^{-i\omega_F t}$$

$$\omega_S = \sqrt{\frac{k}{m}} \rightarrow \text{slow}, \omega_F = \sqrt{\frac{2k + \tilde{k}}{m}} \rightarrow \text{fast}$$

$$\eta_1, \eta_2 \in \mathbb{R} : c_1 = c_2^* = A_S/2 e^{i\varphi_S} : c_3 = c_4^* = A_F/2 e^{i\varphi_F}$$

Guess: $\eta_1 = A_S \cos(\omega_S t + \varphi_S) + A_F \cos(\omega_F t + \varphi_F)$

$$\eta_2 = A_S \cos(\omega_S t + \varphi_S) - A_F \cos(\omega_F t + \varphi_F)$$

Interpretation:

1) If $A_F = 0$: $\eta_1(t) = \eta_2(t) = A_S \cos(\omega_S t + \varphi_S) \rightarrow \text{Periodic cos} \Rightarrow \text{SHO}$

\Rightarrow This describes one normal mode

2) If $A_S = 0$: $\eta_1(t) = -\eta_2(t) = A_F \cos(\omega_F t + \varphi_F) \rightarrow \text{Periodic cos} \Rightarrow \text{SHO}$

\Rightarrow This describes the second normal mode

Eigenvectors: $\begin{pmatrix} 1 \\ 1 \end{pmatrix} \& \begin{pmatrix} 1 \\ -1 \end{pmatrix} \Rightarrow A = \begin{pmatrix} 1 & 1 \\ 1 & -1 \end{pmatrix}$

Recall: $\vec{\xi} = A^T \vec{\eta}$, $\xi_1 = \eta_1 + \eta_2 = 2A_S \cos(\omega_S t + \varphi_S)$ & $\xi_2 = \eta_1 - \eta_2 = 2A_F \cos(\omega_F t + \varphi_F)$

Assignment 8:

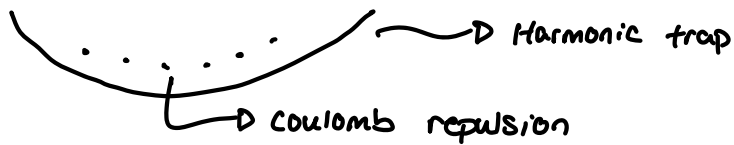
Q1: Rigid Body Motion problem

G_2 : Normal mode analysis

G_3 : Normal mode analysis

qubit \rightarrow Two-level system \rightarrow Described by $\hat{\sigma}_\alpha \rightarrow \sigma'_S = \begin{pmatrix} 1 & 0 \\ 0 & -1 \end{pmatrix}$

Trapped Ions



Normal modes of a crystal \Rightarrow vibrations $\hat{x}_i \rightarrow \hat{\xi}_i$ phonons \rightarrow Two-particle gates

$$\hat{\xi}_i \leftrightarrow \hat{\sigma}_\alpha$$