

ESO 202A/204: Mechanics of Solids (2016-17 II Semester)
Assignment No. - 7 (Solution)

PROB.#1

EQUILIBRIUM:

$$(\sum M_E = 0: 50 - F(0.05) = 0 \Rightarrow F = 1000 \text{ N}$$

$$(\sum M_F = 0: T' - F(0.125) = 0 \\ T' - 1000(0.125) = 0 \Rightarrow T' = 125 \text{ Nm}$$

INTERNAL TORQUE:

As shown in Fig:

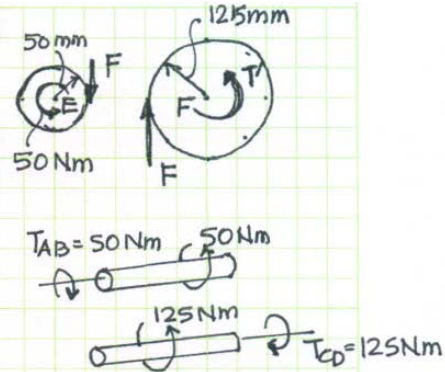
$$T_{AB} = 50 \text{ Nm}$$

$$T_{CD} = 125 \text{ Nm}$$

MAX. SHEAR STRESS:

$$\tau_{AB}|_{\max} = \frac{T_{AB} R_o}{I_p} = \frac{50(0.015)}{\frac{\pi}{2}(0.015^4)} = 9.43 \text{ MPa} \quad \underline{\text{ANS}}$$

$$\tau_{CD}|_{\max} = \frac{T_{CD} R_o}{I_p} = \frac{(125)(0.0175)}{\frac{\pi}{2}(0.0175^4)} = 14.8 \text{ MPa} \quad \underline{\text{ANS}}$$



PROB. #2

EQUILIBRIUM: FBD of AB

$$\sum M_x = 0: -M_t^{AB} - 2000 = 0 \Rightarrow M_t^{AB} = -2000 \text{ Nm}$$

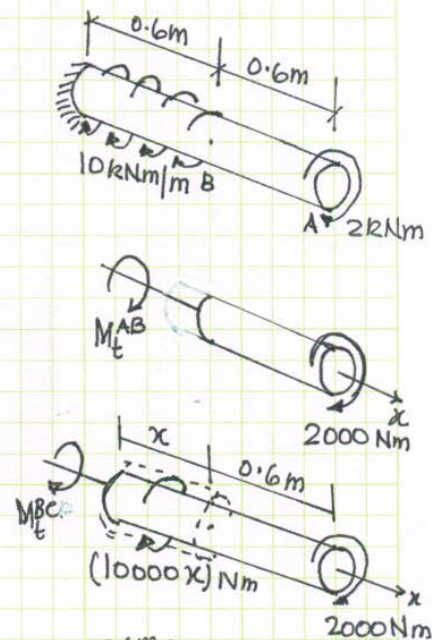
FBD of segment BC

$$\sum M_x = 0: -M_t^{BC} - (10000x) - 2000 = 0 \\ \Rightarrow M_t^{BC} = -[10000x + 2000] \text{ Nm}$$

ANGLE OF TWIST:

$$I_p = \frac{\pi}{2}(0.04)^4 = 1.28 \times 10^{-6} \pi \text{ m}^4$$

$$\phi_A = \sum \frac{M_t^i L_i}{I_p G_i} = \left(\frac{M_t L}{I_p G} \right)_{AB} + \int_0^{0.6} \frac{M_t^{BC}}{(I_p G)_{BC}} dx \\ = \frac{(-2000)(0.6)}{1.28 \times 10^{-6} \pi \times 26 \times 10^9} \\ + \int_0^{0.6} \frac{-[10000x + 2000]}{(1.28 \times 10^{-6}) \pi (26 \times 10^9)} dx \\ = - \frac{1}{(1.28 \times 10^{-6}) \pi (26 \times 10^9)} \left\{ 1200 + (5000x^2 + 2000x) \right\}_0^{0.6} \\ = -0.04017 \text{ rad} = \underline{\underline{2.30^\circ}} \quad \underline{\text{ANS}}$$



PROB. #3

EQUILIBRIUM: Consider FBD as shown in Figs.

$$M_E^A + F(0.1) - 500 = 0 \quad (1)$$

$$M_E^B - F(0.05) = 0 \quad (2)$$

From (1) & (2)

$$M_E^A + 2M_E^B - 500 = 0 \quad (3)$$

COMPATIBILITY:

$$0.1 \phi_E = 0.05 \phi_F$$

$$\Rightarrow \phi_E = 0.5 \phi_F$$

$$\frac{M_E^A (1.5)}{I_P G} = 0.5 \left[\frac{M_E^B (0.75)}{I_P G} \right]$$

$$M_E^A = 0.25 M_E^B \quad (4)$$

Solving (3) & (4) yields

$$M_E^B = 222 \text{ Nm}$$

$$M_E^A = 55.6 \text{ Nm}$$

ANS

