

Q1) Given, Overall Pressure Ratio = 4.0

$$\dot{m} = 3 \text{ kg/s}$$

$$\eta_c = 88\%$$

(ΔT_0) Stagnation pr. rise $< 25 \text{ K}$

(i) $N = ?$ and pr. ratio for first & last stage

we know,

$$\frac{P_{02}}{P_{01}} = \left(1 + N \frac{\Delta T_0}{T_{01}} \right)^{\frac{\gamma}{\gamma-1}} \quad \text{--- (1)}$$

using the above relation, and taking $T_0 = 288 \text{ K}$ (ambient)

$$4.0 = \left(1 + N \times \frac{25}{288} \right)^{\frac{0.28 \times 1.4}{0.4}} \Rightarrow N = \underline{6.54}$$

$\therefore N$ represents no. of stages & \therefore should be an integer ≥ 6.54

Ans \rightarrow $N = 7$; \therefore actual $\Delta T_0 = 23.37 \text{ K}$ (from (1), for $N = 7$)

$$\begin{aligned} T_{02} &= T_{01} + \Delta T \\ &= 311.37 \text{ K} \end{aligned}$$

\therefore Pressure ratio for first stage $\left(\frac{P_{02}}{P_{01}} \right) = \left(\frac{311.37}{288} \right)^{\frac{1.4 \times 0.28}{0.4}}$

$$= \underline{1.27} \quad \leftarrow \underline{Ans}$$

\therefore Pressure ratio for last stage $(T_0)_{in} = 288 + 6(23.37) = 428.22$

$$(T_0)_{out} = 288 + 7(23.37) = 451.59$$

$\therefore \left(\frac{P_{02}}{P_{01}} \right) = \left(\frac{451.6}{428.2} \right)^{\frac{1.4 \times 0.28}{0.4}}$

$$= \underline{1.178} \quad \leftarrow \underline{Ans}$$

(ii) Given, λ (work factor) = 0.83, $\Delta T = 25^\circ\text{K}$

also, $c_p = 1.005 \text{ kJ/kgK}$ & $C_z = 165 \text{ m/s}$

$$\psi = \frac{c_p \Delta T}{u^2} = \frac{\lambda C_z}{u} (\tan \beta_1 - \tan \beta_2) = \lambda [1 - \phi (\tan \beta_2 + \tan \alpha_1)] \quad \because \alpha_1 = \beta_2$$

$$\text{then, } \frac{c_p \Delta T}{\lambda} = u^2 - 2u C_z \tan 20^\circ \quad \left(\because \frac{c_p \Delta T}{u^2} = \left(1 - \frac{2 C_z \tan 20^\circ}{u}\right) \lambda \right)$$

$$\Rightarrow u^2 - 2u C_z \tan 20^\circ - \frac{c_p \Delta T}{\lambda} = 0$$

$$0.83u^2 - 93.68u - 23475.16 = 0 \quad \text{Now solving the eqn}$$

$$\boxed{u = 233.23 \text{ m/s}} = u_2$$

$$\text{Now, } r = 0.09 \text{ m} ; \omega \Rightarrow \frac{2712.4 \times 60}{2\pi} = 413.7 \text{ radian/s}$$

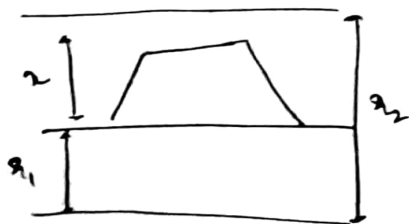
$$\therefore \dot{m} = \int C_z \frac{\pi}{4} (d_2^2 - d_1^2) \quad \text{with } d_2 = 18 \text{ cm}$$

$$\text{where, } \int \Rightarrow \frac{\rho}{\Delta T} = \frac{1.01 \times 10^5}{287 \times 283} = 1.18 \text{ kg/m}^3$$

$$d_2^2 - d_1^2 = \frac{4\dot{m}}{\int C_z \pi} = \frac{4 \times 3}{1.18 \times 165 \times 3.14} = 0.021 \text{ m}^2$$

$$\therefore \boxed{d_1 = 10.7 \text{ cm}}$$

$$\begin{aligned} \text{Now, length of rotor} &= r_2 - r_1 \\ &= 9 - 5.35 \\ &= 3.65 \text{ cm} \end{aligned}$$



$$\begin{aligned} C_2 &= C_1 \cos 20^\circ \\ &= 165 \cos 20^\circ \end{aligned}$$

$$\boxed{C_2 = 155 \text{ m/s}}$$

Q2) Given, $N = 10$; Axial flow compressor $\frac{p_{02}}{p_{01}} = 5:1$
 $\eta_s = 87\%$; $T_{in} = 288 \text{ K}$; identical stages

also, $R = 1/2$, $C_2 = 170 \text{ m/s}$, $u_1 = 210 \text{ m/s}$, $\lambda = 1$; $\beta_1, \beta_2 = ?$

$$\eta_{\text{overall}} \Rightarrow \frac{T_{02s} - T_{01}}{T_{02} - T_{01}} = \frac{T_{01} \left[\left(\frac{p_{02}}{p_{01}} \right)^{\frac{\gamma-1}{\gamma}} - 1 \right]}{T_{02} - T_{01}} = \eta_s = 0.87$$

$$\underbrace{T_{02} - T_{01}}_{10 \text{ stages}} = \frac{288 \left[(5)^{\frac{0.4}{1.4}} - 1 \right]}{0.87} = 193.3 \text{ K}$$

$$\boxed{\Delta T \text{ (single stage)} = 19.3 \text{ K}}$$

Now,

$$\Delta T_s = \frac{\lambda u C_2}{C_p} (\tan \beta_1 - \tan \beta_2)$$

$$(\tan \beta_1 - \tan \beta_2) = \frac{19.3 \times 1.005}{(210) \times (170)} = 0.54$$

Now,

$$R = \frac{1}{2} \phi (\tan \beta_1 + \tan \beta_2) \Rightarrow \left\{ \frac{1}{\phi} = \tan \beta_1 + \tan \beta_2 = \frac{u}{C_2} = 1.23 \right\}$$

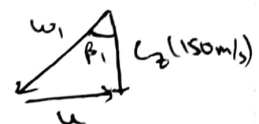
from above two equations,

$$\boxed{\beta_1 = 41.67^\circ} \quad \& \quad \boxed{\beta_2 = 19^\circ}$$

Q3) Given, $p_{in} = 1.0132 \text{ bar}$, $C_2 = 150 \text{ m/s}$ (no IGV)

$d_t = 60 \text{ cm}$, $d_{hub} = 50 \text{ cm}$, $N = 100 \text{ rps}$

for blade speed (u) $\Rightarrow u = 2\pi N r_{mean}$
 $= 2\pi (100) (0.55) \text{ m/s}$
 $= 172.78 \text{ m/s}$



$$\text{then, } w_1 = \sqrt{C_2^2 + u^2} = 228.8 \text{ m/s}$$

$$\tan \beta_1 = u/C_2 \Rightarrow \beta_1 = 49.036^\circ$$

for the complete design,

since air is deflected by 30°

$$\beta_2 = \beta_1 - 30^\circ \Rightarrow \boxed{\beta_2 = 19.036^\circ}$$

$$w_2 = \frac{C_2}{\cos(19.036)} = 158.67 \text{ m/s}$$

$$\text{Now, } C_2 = \sqrt{C_z^2 + (u_2 - w_2 \sin \beta_2)^2} = 192.74 \text{ m/s}$$

$$\text{also, } \alpha_2 = \cos^{-1} \frac{150}{192.74} \Rightarrow \boxed{\alpha_2 = 38.4^\circ}$$

$$\dot{m} = \rho A C_2 \quad \& \quad T_1 \Rightarrow T_0 - \frac{C^2}{2c_p} = 276.8 \text{ K}$$

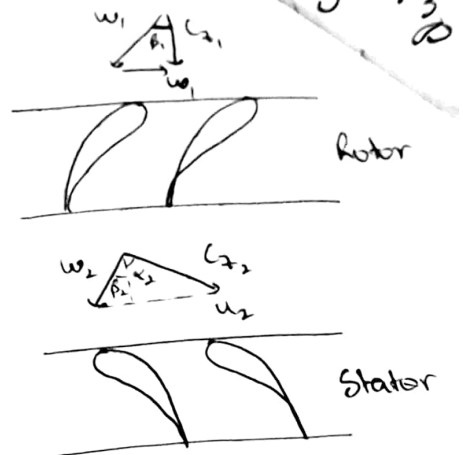
$$\text{with, } P = P_0 \left(\frac{T}{T_0} \right)^{\gamma/\gamma-1} \Rightarrow \boxed{P_1 = 0.8816 \text{ bar}}$$

$$\rho = \frac{P_1}{RT} = 1.110 \text{ kg/m}^3 \Rightarrow \boxed{\dot{m} = 14.38 \text{ kg/s}}$$

$$\text{Power} \Rightarrow \dot{m} (u_2 C_{\theta 2} - u_1 C_{\theta 1}^2) = 14.38 \times 172.78 \times 192.74 \times \sin 38.9^\circ = 300.72 \text{ kW}$$

$$R \Rightarrow \frac{w_{\theta 1} + w_{\theta 2}}{2u} = \frac{u + w_2 \sin \beta_2}{2u}$$

$$\boxed{\therefore R = 0.65}$$



Given, axial flow compressors ; $R = 1/2$; $\beta_1 = 45^\circ$; $\beta_2 = 10^\circ$

Q4) $T_{01} = 310 \text{ K}$; $\frac{P_{02}}{P_{01}} = 6$ & $\eta_s = 0.85$

Assume, $u = 200 \text{ m/s}$

$$\eta_s = \frac{T_{02s} - T_{01}}{T_{02} - T_{01}} = \frac{310 \left[\left(\frac{P_{02}}{P_{01}} \right)^{\frac{\gamma-1}{\gamma}} - 1 \right]}{T_{02} - T_{01}} = 310 \left[6^{\frac{0.4}{1.4}} - 1 \right]$$

$$\Rightarrow T_{02} - T_{01} = 243.8 \text{ K} \text{ for all stages}$$

Now, to find C_z $\beta_1 = 45^\circ$; $\beta_2 = 10^\circ$; $\tan \beta_m \Rightarrow \frac{\tan \beta_1 + \tan \beta_2}{2} = 0.58$

Now, $R = \phi \tan \beta_m = 1/2$

$$\phi = \frac{C_z}{u} = \frac{1}{2 \tan \beta_m} \Rightarrow C_z = \frac{100}{0.588} = 170.06 \text{ m/s}$$

Temp. drop for single stage, $\Delta T = \frac{\lambda u C_z}{C_p} (\tan \beta_1 - \tan \beta_2)$

\therefore for $\lambda = 1$; $\Delta T = 27.87 \text{ K}$ & for $\lambda = 0.87$, $\Delta T = 24.24 \text{ K}$

$$N_1 \Rightarrow \frac{243.8}{27.87} = 8.74 \quad \left| \quad N_2 \Rightarrow \frac{243.8}{24.24} = 10.05 \right.$$

which gives us $N_1 = 9 \text{ stages}$; $N_2 = 10 \text{ stages}$

Q5) Given, head pr. ratio = 4

isentropic eff. = 85%

head inlet temp = 290 K

$\beta_1 = 45^\circ$ & $\beta_2 = 10^\circ$

$R = 1/2$

u & C_z are constant

also, $u = 220 \text{ m/s}$ & $\lambda = 0.86$; $N = ?$

$$\eta_c = \frac{\left(\frac{P_2}{P_1}\right)^{\frac{r-1}{r}} - 1}{\left(\frac{P_2}{P_1}\right)^{\frac{r-1}{r} \eta_p} - 1}$$

where, $\eta_c = 0.85$, $\frac{P_2}{P_1} = 4$, $r = 1.4$

$$\left(\frac{P_2}{P_1}\right)^{\frac{r-1}{r} \eta_p} - 1$$

& η_p = polytropic efficiency

$$\Rightarrow \eta_c = \frac{(4)^{\frac{0.4}{1.4}} - 1}{(4)^{\frac{0.4}{1.4} \eta_p} - 1} = 0.85$$

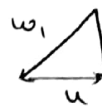
$$\frac{0.485}{0.85} = (4)^{\frac{0.4}{1.4} \eta_p} - 1 \Rightarrow \boxed{\eta_p = 0.866 \text{ or } \eta_p = 86.6\%}$$

Rotor & stator blades are symmetric ($\alpha_1 = \beta_2$, $\beta_1 = \alpha_2$)

$$R = 0.5 \Rightarrow \frac{C_z}{2u} (\tan \beta_1 + \tan \beta_2)$$

$$\frac{220}{(\tan 45^\circ + \tan 10^\circ)} = C_z \quad \left| \quad \boxed{C_z = 187.02 \text{ m/s}} \right|$$

Q7) $P_1 = 0.9 \text{ bar}$
 $T_1 = 288 \text{ K}$

ω_1  $c_1 = c_2$ $\alpha_1 = 0$

Density (ρ) $\Rightarrow \frac{P_1}{RT_1} = \frac{0.9 \times 10^5}{287 \times 288} = 1.09 \text{ kg/m}^3$

Given,

$\dot{Q} = 1000 \text{ m}^3/\text{min}$; Area of airfoil = 19.25 cm^2

(Length)_{blade} = 6.75 cm ; $N = 6000 \text{ rpm}$; $D_m = 60 \text{ cm}$

\therefore no. of blades = 50 (16% volume); $C_z = 0.6$ & $C_b = 0.05$

$$u \Rightarrow \frac{\pi D_m N}{60}$$

$$\Rightarrow 188.5 \text{ m/s}$$

$$C_z \Rightarrow \dot{Q}/A$$

$$\Rightarrow \frac{1000 \times 10^4}{60(1.01) \times 60 \times 6.75}$$

$$\Rightarrow \boxed{C_z = 145.6 \text{ m/s}}$$

$$\tan \beta_1 \Rightarrow \frac{u}{c_2} = \frac{188.5}{145.6} = 1.295 \Rightarrow \boxed{\beta_1 = 52.3^\circ}$$

$$\boxed{w_1 = \sqrt{145.6^2 + 188.5^2} = 238.15 \text{ m/s}}$$

$$\text{Lift} \Rightarrow \frac{C_L \rho w_1^2 A_c}{2} = \frac{0.6 \times 1.09 \times (238.15)^2 \times 19.25 \times 10^{-4}}{2} = 35.7 \text{ N}$$

$$\text{Drag} \Rightarrow \frac{C_D \rho w_1^2 A_c}{2} = \frac{0.05 \times 1.09 \times (238.15)^2 \times 19.25 \times 10^{-4}}{2} = 2.98 \text{ N}$$

Now,

$$\text{Power input per stage} \Rightarrow (L \cos \beta_1 + D \sin \beta_1) \text{ uN}$$

$$\Rightarrow [35.7 \cos(52.35) + 2.9 \sin(52.35)] (188.5 \times 50 \times 10^{-3})$$

$$\Rightarrow \boxed{227.85 \text{ kW}}$$

$$\dot{m} = \dot{Q} \rho = \frac{9000 \times 1.09}{60} = 18.167 \text{ kg/s}$$

Now,

$$\frac{P}{\dot{m}} = \frac{C_p T_1}{\eta_c} \left[r^{\frac{\gamma-1}{\gamma}} - 1 \right]$$

$$\Rightarrow r^{\frac{0.4}{1.4}} \Rightarrow \frac{P}{\dot{m}} \times \frac{\eta_c}{C_p T_1} + 1 \Rightarrow \left[\frac{227.85 \times 1}{18.33 \times 1.005 \times 288} - 1 \right]$$

$$\text{or } \boxed{r = 1.16 = \frac{P_2}{P_1}}$$

$$\boxed{\therefore \Delta P_{\text{stage}} = 1.14 \text{ bar}} \quad \underline{\underline{Ans}}$$