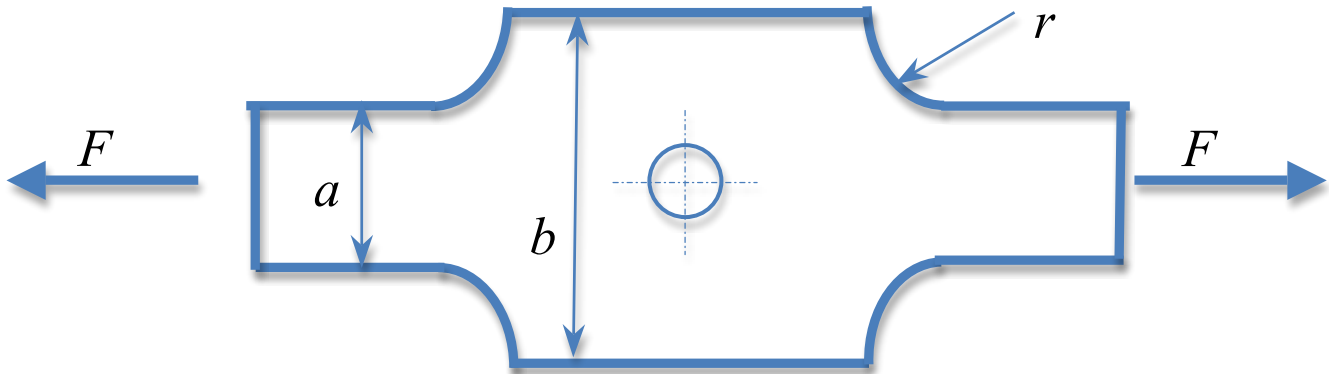


## Tutorial 2

### ME-351, 2013-14 2<sup>nd</sup> Semester

January 23, 2014

#### Problem 1:



The connecting-link shown above is subject to an axial force  $F$ . It has a thickness  $h$  and the other dimensions are:  $a = 60\text{mm}$ ,  $b = 100\text{mm}$ ,  $r = 6\text{mm}$ , hole diameter =  $20\text{mm}$ . The material of the link is cold-drawn AISI 1018 steel. The link is subject to a completely reversible load of magnitude  $80\text{kN}$ .

- (a) Determine the critical location of the link.
- (b) Determine the thickness of the link to have a factor of safety of at least 3 against yielding. Choose a standard value for the thickness.
- (c) Recalculate the thickness such that the connecting-link has infinite life with a fatigue factor of safety of at least 2.5. Choose a standard value for the thickness.
- (d) For the thickness calculated in (c), determine the new load level if you want to have a finite life of 10,000 cycles.
- (e) Determine the fatigue factor of safety if the load changes from completely reversible one and fluctuates between a tension of  $80\text{kN}$  and compression of  $20\text{kN}$ . Use modified Goodman failure criteria (ignore column buckling).

**Problem 2:** A cantilever beam with circular cross section of diameter  $d$  and length  $250\text{mm}$  is subject to a load of  $5\text{kN}$  at the tip and a torque of  $2\text{kN}\cdot\text{m}$ . Determine the diameter  $d$  if you want the beam to have a factor of safety of 2.5 against yielding using

- (a) Tresca failure criteria
- (b) Von Mises failure criteria and
- (c) Mohr-Coulomb failure criteria.

Finally choose a standard size for the beam. Use AISI 1020 hot-rolled steel as beam material.