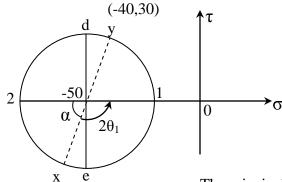
## ESO 202A/204: MECHANICS OF SOLIDS (2016-17 II Semester) Assignment No. 3 Answer Sheet

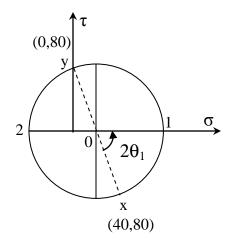




radius =  $\sqrt{30^2 + 10^2}$  = 31.6 MPa  $\sigma_d = \sigma_e = -50$  MPa  $\sigma_1 = -50 + 31.6 = -18.4$  MPa  $\sigma_2 = -50 - 31.6 = -81.6$  MPa  $\tan \alpha = \frac{30}{10} \Rightarrow \alpha = 71^{\circ}33'$   $2\theta_1 = 108^{\circ}27' \Rightarrow \theta_1 = 54^{\circ}13.5'$ Maximum shearing stress = 31.6 MPa

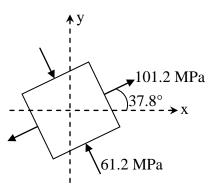
The principal planes are 54°13.5′ and 144°13.5′ Planes of maximum shear are 9°13.5′ and 99°13.5′

## 3.2

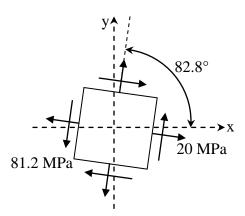


(-60,30)

radius = 
$$\sqrt{20^2 + 80^2}$$
 = 81.2 MPa  
 $\sigma_1 = 20 + 81.2 = 101.2$  MPa  
 $\sigma_2 = 20 - 81.2 = -61.2$  MPa  
 $\tau_{max} = 81.2$  MPa  
 $\tan 2\theta_1 = \frac{80}{20} = 4 \Rightarrow 2\theta_1 = 75.6^\circ \Rightarrow \theta_1 = 37.8^\circ$ 

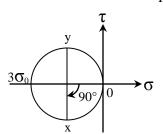


Element oriented along Principal axes



Element oriented along maximum shear axes

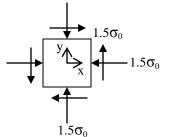
**3.3** First resolve the second part along x-y directions and superimpose on the first part.

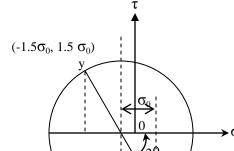


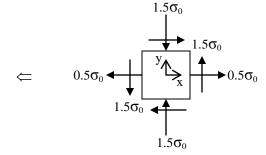
$$\sigma_{xx} = -1.5\sigma_0$$

$$\sigma_{yy} = -1.5 \sigma_0$$

$$\tau_{xy} = 1.5\sigma_0$$





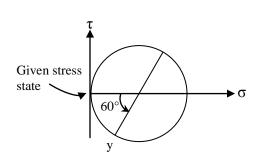


 $\downarrow \downarrow$ 

Principal Direction:

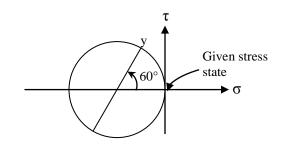
$$\theta_1 = \frac{1}{2} \tan^{-1} \left( \frac{1.5\sigma_0}{\sigma_0} \right)$$
$$= 28.15^{\circ}$$

**3.4** Maximum shear stress = radius = 500 MPa



OR

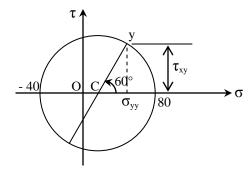
 $(0.5\sigma_0, 1.5 \sigma_0)$ 



$$\sigma_{yy}=\pm~500~cos60^\circ=\pm~250~MPa$$

$$\tau_{xy} = \pm 500 \sin 60^{\circ} = \pm 433 \text{ MPa}$$

3.5 Since the two given planes are perpendicular and there are no shear stresses on these planes, the given stresses are the principal stresses.



$$\sigma_2 = -40$$

$$30^{\circ}$$

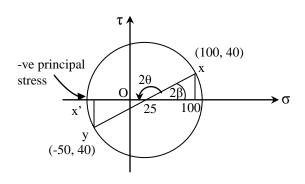
$$60^{\circ}$$

$$\sigma_{yy}=20+30=50~MPa$$

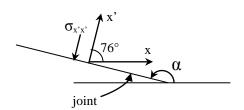
$$\tau_{yy} = 60 \ (\sqrt{3}/2) = 30\sqrt{3} \text{ MPa}$$

Maximum Shear Stress = 60 MPa

3.6 Since the joint cannot take any shear or tension, the inclination  $\alpha$ , ( $2\alpha$  in Mohr's circle) will be such that the principal stress perpendicular to the joint is negative.



$$2\beta = \tan^{-1} (40/75) = 28^{\circ}$$
  
 $2\theta = 180^{\circ} - 28^{\circ} = 152^{\circ}$   
 $\theta = 76^{\circ}$ 



Obviously, 
$$\alpha = 90 + 76 = 166^{\circ}$$
  
 $\sigma_{yy} = -50 \text{ MPa}$ 

3.7 The centre of Mohr's circle C should be on  $\sigma$ -axis such that the angle between Cn<sub>1</sub> and Cn<sub>2</sub> should be  $240^0$  in CW sense.

Draw the Mohr's circle.

$$OD = 2\sigma_0$$

$$Dn_1 = \sqrt{3} \sigma_0$$

$$tan60^0 = \sqrt{3} = \frac{Dn_1}{CD}$$
or,
$$CD = \frac{\sqrt{3}\sigma_0}{\sqrt{3}}$$

$$CD = OD + CD = 3\sigma_0$$
Also,

$$Cn_{1} = \left[\sigma_0^2 + (\sqrt{3}\sigma_0)^2\right]^{\frac{1}{2}}$$
$$= 2\sigma_0$$

· Principal stresses are,

$$\sigma_1 = OC + radius(Cn_1)$$

$$= 3\sigma_0 + 2\sigma_0 = 5\sigma_0 \quad [Ans]$$

$$\sigma_2 = 3\sigma_0 - 2\sigma_0 = \sigma_0$$

