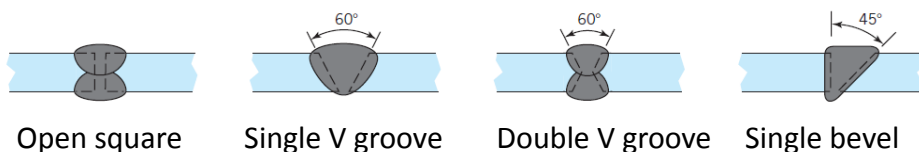


## Welded joints

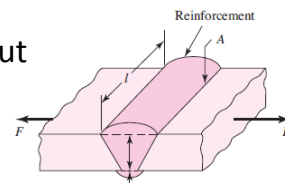
- Welding process involving fusion of the work pieces together
- Extensively used in steel structures
- Compared to bolted or riveted joints welding
  - does not require geometry changes in members (holes)
  - is leak proof
  - quicker and less in weight
- Members are taken to higher temperature
  - can lead to metallurgical changes that can affect their properties locally: **Heat Affected Zone**
  - can induce local residual stresses and distortion
- Is permanent connection
- Properties of welding rod (filler material) should be matched properly with those of parent material

## Butt welds



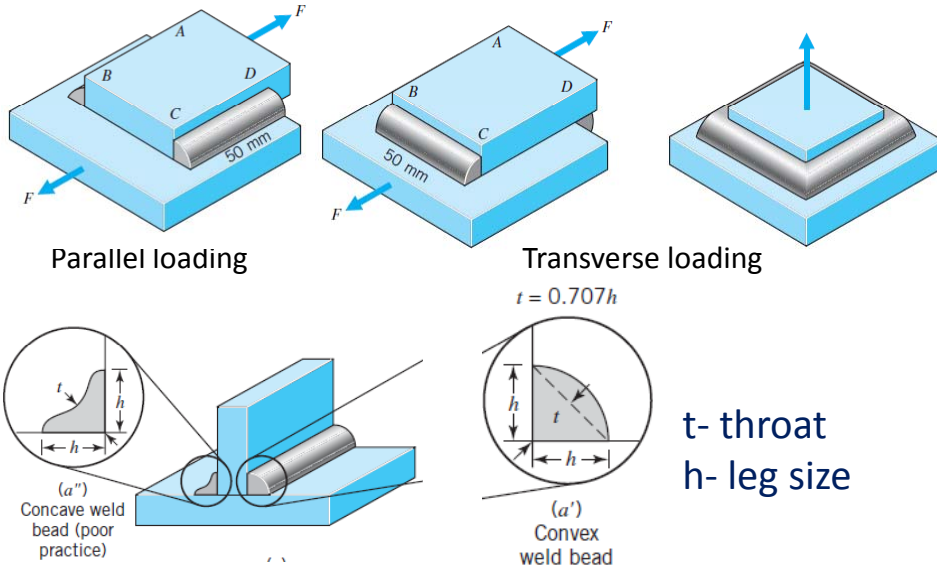
Open square      Single V groove      Double V groove      Single bevel

- If the strength of the electrode is matched with that of the parent material, then no analysis is needed for static loads
- If filler penetrates fully through the thickness and welding is properly done, then joint is as strong as the parent material
- In fatigue loading, voids and inclusions can cause stress concentration
- The “reinforcement” i.e weld protruding out beyond plate thickness can cause stress concentration (point A in figure)
- See table 9-5 in data book



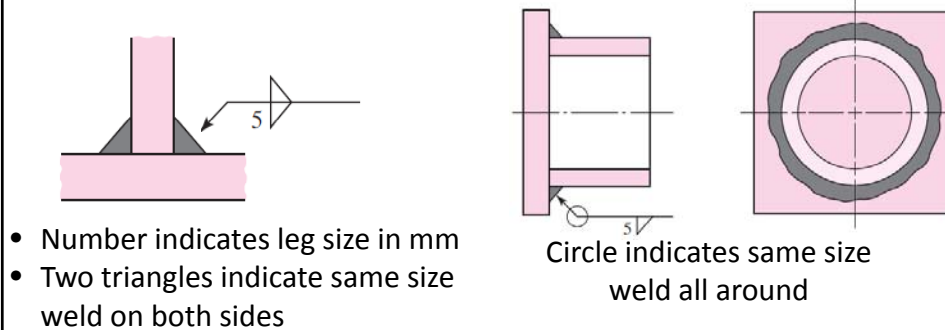
## Fillet welds

- Classified according to direction of loading

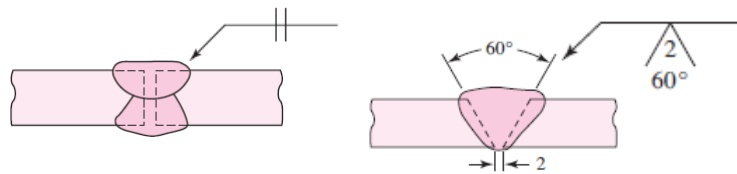


## Welding symbols

### Fillet welds

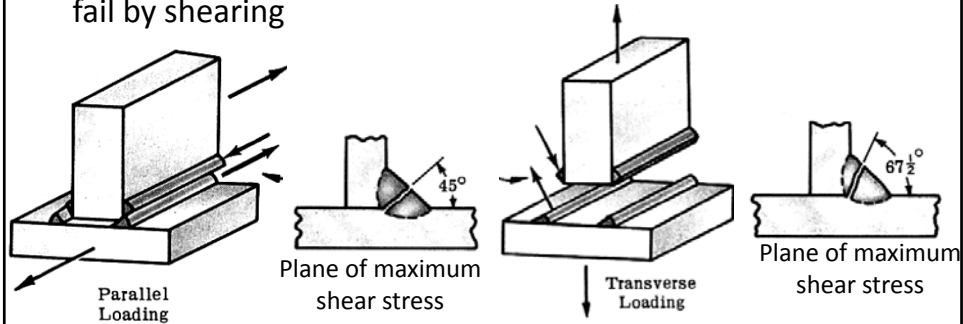


### Butt welds

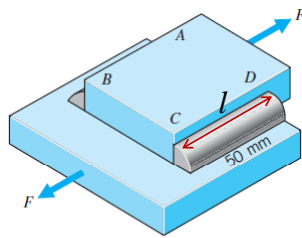


## Design of fillet welds

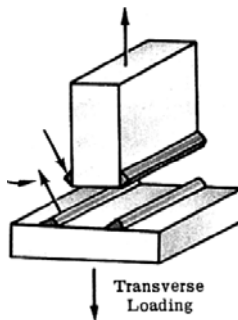
- Irrespective of type of loading, fillet welds are observed to fail by shearing



- For same length and leg size, strength will be more in transverse loading as area along plane of maximum shear stress (67.5°) is higher
- Design is carried out using the area along the throat (is 45° plane)
  - Takes care of uncertainty of loading direction
  - Conservative



$$\tau = \frac{F}{A_{throat}} = \frac{F}{2(0.707hl)} = \frac{0.707F}{hl}$$

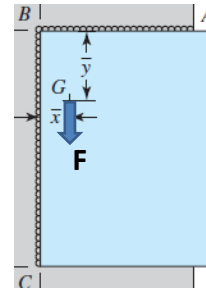
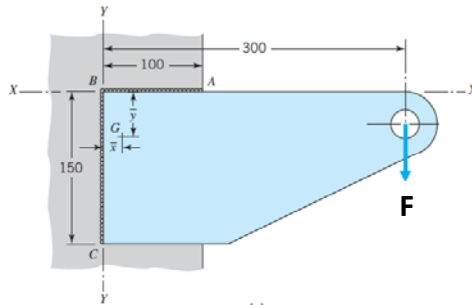


A diagram of a fillet weld joint under transverse loading, showing the plane of maximum shear stress at 67.5°. The formula  $\tau = \frac{0.605F}{hl}$  is shown with a large red 'X' over it, indicating it is not to be used.

$$\tau = \frac{0.707F}{hl}$$

Use this expression

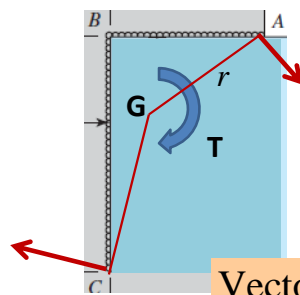
## Weld under torsion



- Determine the centroid

Primary shear;  $\tau' = \frac{F}{A_{throat}}$

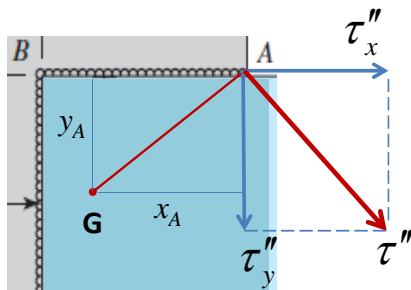
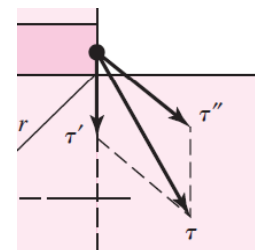
- Assumed to be uniformly distributed over the entire length of the weld



Secondary shear;  $\tau'' = \frac{Tr}{J}$

$J$  is the polar moment of inertia of throat area



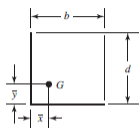
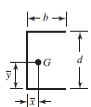
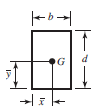

Vectorially add  $\tau'$  and  $\tau''$



$$\tau''_x = \frac{Ty_A}{J}; \quad \tau''_y = \frac{Tx_A}{J}$$

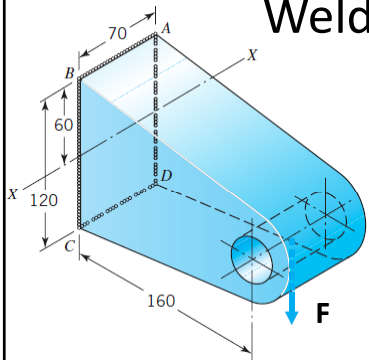
$$\tau'_x = 0; \quad \tau'_y = \frac{F}{A_{throat}}$$

$$\tau = \left\{ (\tau'_x + \tau''_x)^2 + (\tau'_y + \tau''_y)^2 \right\}^{1/2}$$

| Weld  | Throat Area           | Location of G  | Unit Second Polar Moment of Area                         |
|---|-----------------------|--|--|
|  | $A = 0.707 h d$       | $\bar{x} = 0$<br>$\bar{y} = d/2$                                 | $J_u = d^3/12$   |
|  | $A = 1.41 h d$        | $\bar{x} = b/2$<br>$\bar{y} = d/2$                               | $J_u = \frac{d(3b^2 + d^2)}{6}$                          |
|  | $A = 0.707 h(2b + d)$ | $\bar{x} = \frac{b^2}{2(b+d)}$<br>$\bar{y} = \frac{d^2}{2(b+d)}$ | $J_u = \frac{[b+d]^4 - 6b^2d^2}{12(b+d)}$                |
|  | $A = 0.707 h(2b + d)$ | $\bar{x} = \frac{b^2}{2b+d}$<br>$\bar{y} = d/2$                  | $J_u = \frac{8b^3 + 6bd^2 + d^3}{12} - \frac{b^4}{2b+d}$ |
|  | $A = 1.414 h(b + d)$  | $\bar{x} = b/2$<br>$\bar{y} = d/2$                               | $J_u = \frac{[b+d]^3}{6}$                                |
|  | $A = 1.414 \pi r$     |  | $J_u = 2\pi r^3$   |

$$J = 0.707 h J_u$$

### Weld under bending



Primary shear;  $\tau' = \frac{F}{A_{throat}}$

Secondary shear;  $\tau'' = \frac{My}{I}$

$$\tau = \sqrt{(\tau')^2 + (\tau'')^2}$$

| Weld | Throat Area          | Location of $G$                                   | Unit Second Moment of Area                               |
|------|----------------------|---|--|
|      | $A = 0.707hd$        | $\bar{x} = 0$<br>$\bar{y} = d/2$                  | $I_y = \frac{d^3}{12}$                                   |
|      | $A = 1.414hd$        | $\bar{x} = b/2$<br>$\bar{y} = d/2$                | $I_y = \frac{d^3}{6}$                                    |
|      | $A = 1.414hd$        | $\bar{x} = b/2$<br>$\bar{y} = d/2$                | $I_y = \frac{bd^2}{2}$                                   |
|      | $A = 0.707h(2b + d)$ | $\bar{x} = \frac{b^2}{2b + d}$<br>$\bar{y} = d/2$ | $I_y = \frac{d^2}{12}(6b + d)$                           |
|      | $A = 0.707h(b + 2d)$ | $\bar{x} = b/2$<br>$\bar{y} = \frac{d^2}{b + 2d}$ | $I_y = \frac{2d^3}{3} - 2d^2\bar{y} + (b + 2d)\bar{y}^2$ |
|      | $A = 1.414h(b + d)$  | $\bar{x} = b/2$<br>$\bar{y} = d/2$                | $I_y = \frac{d^2}{6}(3b + d)$                            |

**$I = 0.707hI_u$**   
I is about horizontal axis

## Strength of weld electrode

| AWS Electrode Number* | Tensile Strength<br>kpsi (MPa) | Yield Strength,<br>kpsi (MPa) | Percent<br>Elongation |
|-----------------------|--------------------------------|-------------------------------|-----------------------|
| E60xx                 | 62 (427)                       | 50 (345)                      | 17-25                 |
| E70xx                 | 70 (482)                       | 57 (393)                      | 22                    |
| E80xx                 | 80 (551)                       | 67 (462)                      | 19                    |
| E90xx                 | 90 (620)                       | 77 (531)                      | 14-17                 |
| E100xx                | 100 (689)                      | 87 (600)                      | 13-16                 |
| E120xx                | 120 (827)                      | 107 (737)                     | 14                    |

- Yield strength in shear  $S_{sy} = 0.577 S_y$
- Check the design for failure in the weld
- Also check the members based on their material strength (weld attachment)

## Fatigue loading

- SN diagram

$$S_e = k_a k_b k_c k_d k_e k_f S'_e; \quad S'_e = 0.5 S_{ut}$$

- For surface factor , use “as forged” surface finish
- Size factor is 1 as stress is assumed to be uniform
- Load factor is 0.59 since design is based on shear stress
- Stress concentration factor is given in table 9-5
- Apply this on the stress
- Use Goodman line