### **Homework-13 Solutions**

# Q 15-20

#### **Solution:**

Assumptions 1 Combustion is complete. 2 The combustion products contain CO<sub>2</sub>, H<sub>2</sub>O, O<sub>2</sub>, and N<sub>2</sub> only.

**Properties** The molar masses of C, H<sub>2</sub>, O<sub>2</sub>, and air are 12 kg/kmol, 2 kg/kmol, 32 kg/kmol, and 29 kg/kmol, respectively (Table A-1).

Analysis The combustion equation in this case can be written as

$$C_4H_{10} + 2.0a_{th}[O_2 + 3.76N_2] \longrightarrow 4CO_2 + 5H_2O + 1.0a_{th}O_2 + (2.0 \times 3.76)a_{th}N_2$$

where  $a_{th}$  is the stoichiometric coefficient for air. We have automatically accounted for the 100% excess air by using the factor  $2.0a_{th}$  instead of  $a_{th}$  for air. The stoichiometric amount of oxygen  $(a_{th}O_2)$  will be used to oxidize the fuel, and the remaining excess amount  $(1.0a_{th}O_2)$  will appear in the products as free oxygen. The coefficient  $a_{th}$  is determined from the  $O_2$  balance,

O<sub>2</sub> balance: 
$$2.0a_{th} = 4 + 2.5 + 1.0a_{th} \longrightarrow a_{th} = 6.5$$

Substituting, 
$$C_4H_{10} + 13[O_2 + 3.76N_2] \longrightarrow 4CO_2 + 5H_2O + 6.5O_2 + 48.88N_2$$

The mole fractions of the products are

$$N_m = 4 + 5 + 6.5 + 48.88 = 64.38 \text{ kmol}$$

$$y_{CO2} = \frac{N_{CO2}}{N_m} = \frac{4 \text{ kmol}}{64.38 \text{ kmol}} = \mathbf{0.0621}$$

$$y_{H2O} = \frac{N_{H2O}}{N_m} = \frac{5 \text{ kmol}}{64.38 \text{ kmol}} = \mathbf{0.0777}$$

$$y_{O2} = \frac{N_{O2}}{N_m} = \frac{6.5 \text{ kmol}}{64.38 \text{ kmol}} = \mathbf{0.1010}$$

$$y_{N2} = \frac{N_{N2}}{N_m} = \frac{48.88 \text{ kmol}}{64.38 \text{ kmol}} = \mathbf{0.7592}$$

The mass of carbon dioxide in the products per unit mass of fuel burned is

$$\frac{m_{\rm CO2}}{m_{\rm C4H10}} = \frac{(4 \times 44) \,\text{kg}}{(1 \times 58) \,\text{kg}} = 3.034 \,\text{kg CO}_2/\text{kg C}_4\text{H}_{10}$$

The air-fuel ratio is determined by taking the ratio of the mass of the air to the mass of the fuel,

AF = 
$$\frac{m_{\text{air}}}{m_{\text{fuel}}} = \frac{(13 \times 4.76 \text{ kmol})(29 \text{ kg/kmol})}{(1 \text{ kmol})(58 \text{ kg/kmol})} = 30.94 \text{ kg air/kg fuel}$$

## Q 15-28

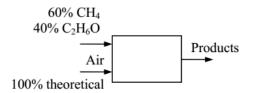
### **Solution:**

Assumptions 1 Combustion is complete. 2 The combustion products contain CO<sub>2</sub>, H<sub>2</sub>O, and N<sub>2</sub> only.

**Properties** The molar masses of C, H<sub>2</sub>, O<sub>2</sub> and air are 12 kg/kmol, 2 kg/kmol, 32 kg/kmol, and 29 kg/kmol, respectively (Table A-1).

Analysis For 100 kg of fuel mixture, the mole numbers are

$$\begin{split} N_{\rm CH4} &= \frac{\rm mf_{\rm CH4}}{M_{\rm CH4}} = \frac{60 \, \rm kg}{16 \, \rm kg/kmol} = 3.75 \, \rm kmol \\ N_{\rm C2H6O} &= \frac{\rm mf_{\rm C2H6O}}{M_{\rm C2H6O}} = \frac{40 \, \rm kg}{46 \, \rm kg/kmol} = 0.8696 \, \rm kmol \end{split}$$



The mole fraction of methane and ethanol in the fuel mixture are

$$x = \frac{N_{\text{CH4}}}{N_{\text{CH4}} + N_{\text{C2H6O}}} = \frac{3.75 \text{ kmol}}{(3.75 + 0.8696) \text{ kmol}} = 0.8118$$
$$y = \frac{N_{\text{C2H6O}}}{N_{\text{CH4}} + N_{\text{C2H6O}}} = \frac{0.8696 \text{ kmol}}{(3.75 + 0.8696) \text{ kmol}} = 0.1882$$

The combustion equation in this case can be written as

$$x \text{ CH}_4 + y \text{ C}_2 \text{H}_6 \text{O} + a_{\text{th}} [\text{O}_2 + 3.76 \text{N}_2] \longrightarrow B \text{ CO}_2 + D \text{ H}_2 \text{O} + F \text{ N}_2$$

where  $a_{th}$  is the stoichiometric coefficient for air. The coefficient  $a_{th}$  and other coefficients are to be determined from the mass balances

Carbon balance: x + 2y = B

Hydrogen balance: 4x + 6y = 2D

Oxygen balance:  $2a_{th} + y = 2B + D$ 

Nitrogen balance:  $3.76a_{th} = F$ 

Substituting x and y values into the equations and solving, we find the coefficients as

$$x = 0.8118$$
  $B = 1.188$   
 $y = 0.1882$   $D = 2.188$   
 $a_{th} = 2.188$   $F = 8.228$ 

Then, we write the balanced reaction equation as

$$0.8118\,\mathrm{CH_4} + 0.1882\,\mathrm{C_2H_6O} + 2.188\,\mathrm{\big[O_2 + 3.76N_2\big]} \longrightarrow 1.188\,\mathrm{CO_2} + 2.188\,\mathrm{H_2O} + 8.228\,\mathrm{N_2}$$

The air-fuel ratio is determined by taking the ratio of the mass of the air to the mass of the fuel,

$$AF = \frac{m_{\text{air}}}{m_{\text{fuel}}} = \frac{(2.188 \times 4.76 \text{ kmol})(29 \text{ kg/kmol})}{(0.8118 \text{ kmol})(12 + 4 \times 1) \text{kg/kmol} + (0.1882 \text{ kmol})(2 \times 12 + 6 \times 1 + 16) \text{kg/kmol}}$$
$$= 13.94 \text{ kg air/kg fuel}$$

Then, the required flow rate of air becomes

$$\dot{m}_{\rm air} = AF\dot{m}_{\rm fuel} = (13.94)(10 \,\text{kg/s}) = 139.4 \,\text{kg/s}$$

## Q 15-60

### **Solution:**

Assumptions 1 Steady operating conditions exist. 2 Air and combustion gases are ideal gases. 3 Kinetic and potential energies are negligible. 4 Combustion is complete.

*Analysis* The fuel is burned completely with the stoichiometric amount of air, and thus the products will contain only  $H_2O$ ,  $CO_2$  and  $N_2$ , but no free  $O_2$ . Considering 1 kmol of fuel, the theoretical combustion equation can be written as

$$CH_4 + a_{th}(O_2 + 3.76N_2) \longrightarrow CO_2 + 2H_2O + 3.76a_{th}N_2$$

where  $a_{th}$  is determined from the  $O_2$  balance,
$$a_{th} = 1 + 1 = 2$$

Substituting,

 $CH_4$ 

$$25^{\circ}C$$

Combustion chamber
$$CH_4$$

$$25^{\circ}C$$

Products
$$CH_4$$

$$25^{\circ}C$$

The heat transfer for this combustion process is determined from the energy balance  $E_{\rm in}-E_{\rm out}=\Delta E_{\rm system}$  applied on the combustion chamber with W=0. It reduces to

$$-\mathcal{Q}_{\mathrm{out}} = \sum N_P \Big( \overline{h}_f^{\,\circ} + \overline{h} - \overline{h}^{\,\circ} \Big)_{\!P} - \sum N_R \Big( \overline{h}_f^{\,\circ} + \overline{h} - \overline{h}^{\,\circ} \Big)_{\!R} = \sum N_P \overline{h}_{f,P}^{\,\circ} - \sum N_R \overline{h}_{f,R}^{\,\circ}$$

 $CH_4 + 2(O_2 + 3.76N_2) \longrightarrow CO_2 + 2H_2O + 5.64N_2$  100% theoretical

since both the reactants and the products are at 25°C and both the air and the combustion gases can be treated as ideal gases From the tables,

Cubatanas	$\overline{h}_f^\circ$
Substance	kJ/kmol
CH <sub>4</sub>	-74,850
$O_2$	0
$N_2$	0
$H_2O(\ell)$	-285,830
$CO_2$	-393,520

Thus,

$$-Q_{\text{out}} = (1)(-393,520) + (2)(-285,830) + 0 - (1)(-74,850) - 0 - 0 = -890,330 \text{ kJ/kmol CH}_4$$

or

$$Q_{\text{out}} = 890,330 \text{ kJ/kmol CH}_4$$

If combustion is achieved with 100% excess air, the answer would still be the same since it would enter and leave at 25°C, and absorb no energy.

### **Solution:**

Assumptions 1 Combustion is complete. 2 The combustion products contain CO<sub>2</sub>, CO, H<sub>2</sub>O, SO<sub>2</sub>, and N<sub>2</sub>. 3 Combustion gases are ideal gases.

Properties The molar masses of C, H2, N2, O2, S, and air are 12, 2, 28, 32, 32, and 29 kg/kmol, respectively (Table A-1).

Analysis We consider 100 kg of coal for simplicity. Noting that the mass percentages in this case correspond to the masses of the constituents, the mole numbers of the constituent of the coal are determined to be

$$N_{\rm C} = \frac{m_{\rm C}}{M_{\rm C}} = \frac{39.25 \,\text{kg}}{12 \,\text{kg/kmol}} = 3.271 \,\text{kmol}$$

$$N_{\rm H2} = \frac{m_{\rm H2}}{M_{\rm H2}} = \frac{6.93 \,\text{kg}}{2 \,\text{kg/kmol}} = 3.465 \,\text{kmol}$$

$$N_{\rm O2} = \frac{m_{\rm O2}}{M_{\rm O2}} = \frac{41.11 \,\text{kg}}{32 \,\text{kg/kmol}} = 1.285 \,\text{kmol}$$

$$N_{\rm N2} = \frac{m_{\rm N2}}{M_{\rm N2}} = \frac{0.72 \,\text{kg}}{28 \,\text{kg/kmol}} = 0.0257 \,\text{kmol}$$

$$N_{\rm S} = \frac{m_{\rm S}}{M_{\rm S}} = \frac{0.79 \,\text{kg}}{32 \,\text{kg/kmol}} = 0.0247 \,\text{kmol}$$

The mole number of the mixture and the mole fractions are

$$N_m = 3.271 + 3.465 + 1.285 + 0.0257 + 0.0247 = 8.071 \text{ kmol}$$

$$y_{\rm C} = \frac{N_{\rm C}}{N_m} = \frac{3.271 \,\text{kmol}}{8.071 \,\text{kmol}} = 0.4052$$

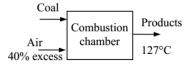
$$y_{\rm H2} = \frac{N_{\rm H2}}{N_m} = \frac{3.465 \,\text{kmol}}{8.071 \,\text{kmol}} = 0.4293$$

$$y_{\rm O2} = \frac{N_{\rm O2}}{N_m} = \frac{1.285 \,\text{kmol}}{8.071 \,\text{kmol}} = 0.1592$$

$$y_{\rm N2} = \frac{N_{\rm N2}}{N_m} = \frac{0.0257 \,\text{kmol}}{8.071 \,\text{kmol}} = 0.00319$$

$$y_{\rm S} = \frac{N_{\rm S}}{N_m} = \frac{0.0247 \,\text{kmol}}{8.071 \,\text{kmol}} = 0.00306$$

39.25% C 6.93% H<sub>2</sub> 41.11% O<sub>2</sub> 0.72% N<sub>2</sub> 0.79% S 11.20% ash (by mass)



Ash consists of the non-combustible matter in coal. Therefore, the mass of ash content that enters the combustion chamber is equal to the mass content that leaves. Disregarding this non-reacting component for simplicity, the combustion equation may be written as

$$\begin{aligned} 0.4052\text{C} + 0.4293\text{H}_2 + 0.1592\text{O}_2 + 0.00319\text{N}_2 + 0.00306\text{S} + 1.4a_{\text{th}}(\text{O}_2 + 3.76\text{N}_2) \\ &\longrightarrow 0.4052\text{CO}_2 + 0.4293\text{H}_2\text{O} + 0.4a_{\text{th}}\text{O}_2 + 0.00306\text{SO}_2 + 1.4a_{\text{th}} \times 3.76\text{N}_2 \end{aligned}$$

According to the O2 mass balance,

$$0.1592 + 1.4a_{th} = 0.4052 + 0.5 \times 0.4293 + 0.4a_{th} + 0.00306 \longrightarrow a_{th} = 0.4637$$

Substituting

$$0.4052C + 0.4293H_2 + 0.1592O_2 + 0.00319N_2 + 0.00306S + 0.6492(O_2 + 3.76N_2)$$

$$\longrightarrow 0.4052CO_2 + 0.4293H_2O + 0.1855O_2 + 0.00306SO_2 + 2.441N_2$$

The heat transfer for this combustion process is determined from the energy balance  $E_{\rm in} - E_{\rm out} = \Delta E_{\rm system}$  applied on the combustion chamber with W = 0. It reduces to

$$-Q_{\mathrm{out}} = \sum N_P \Big( \overline{h}_f^{\,\circ} + \overline{h} - \overline{h}^{\,\circ} \Big)_P - \sum N_R \Big( \overline{h}_f^{\,\circ} + \overline{h} - \overline{h}^{\,\circ} \Big)_R$$

Assuming the air and the combustion products to be ideal gases, we have h = h(T). From the tables,

	$\overline{h}_f^{\circ}$	$\overline{h}_{298~\mathrm{K}}$	$\overline{h}_{ m 400~K}$
Substance	kJ/kmol	kJ/kmol	kJ/kmol
$O_2$	0	8682	11,711
$N_2$	0	8669	11,640
$H_2O(g)$	-241,820	9904	13,356
$CO_2$	-393,520	9364	13,372
$SO_2$	-297,100	-	-

The enthalpy change of sulfur dioxide between the standard temperature and the product temperature using constant specific heat assumption is

$$\Delta \overline{h}_{SO2} = c_p \Delta T = (41.7 \text{ kJ/kmol} \cdot \text{K})(127 - 25)\text{K} = 4253 \text{ kJ/kmol}$$

Substituting into the energy balance relation,

$$\begin{split} -\overline{Q}_{\text{out}} &= \big(0.4052\big)\!\big(\!-393,\!520+\!13,\!372-\!9364\big)\!+ \big(0.4293\big)\!\big(\!-241,\!820+\!13,\!356-\!9904\big) \\ &+ \big(0.1855\big)\!\big(0+\!11,\!711-\!8682\big)\!+ \big(2.441\big)\!\big(0+\!11,\!640-\!8669\big)\!+ \big(0.00306\big)\!\big(\!-297,\!100+\!4253\big)\!- 0 \\ &= -253,\!244 \text{ kJ/kmol C}_8 \text{H}_{18} \end{split}$$

or 
$$\overline{Q}_{\text{out}} = 253,244 \text{ kJ/kmol fuel}$$

Then the heat transfer per kg of fuel is

$$\begin{split} Q_{\text{out}} &= \frac{\overline{Q}_{\text{out}}}{M_{\text{fuel}}} = \frac{253,244 \text{ kJ/kmol fuel}}{(0.4052 \times 12 + 0.4293 \times 2 + 0.1592 \times 32 + 0.00319 \times 28 + 0.00306 \times 32) \text{ kg/kmol}} \\ &= \frac{253,244 \text{ kJ/kmol fuel}}{11.00 \text{ kg/kmol}} \\ &= \mathbf{23,020 \text{ kJ/kg coal}} \end{split}$$

### **Solution:**

Assumptions 1 Steady operating conditions exist. 2 Air and combustion gases are ideal gases. 3 Kinetic and potential energies are negligible. 4 There are no work interactions. 5 The combustion chamber is adiabatic.

Analysis Under steady-flow conditions the energy balance  $E_{\rm in} - E_{\rm out} = \Delta E_{\rm system}$  applied on the combustion chamber with Q = W = 0 reduces to

$$\sum N_P \left(\overline{h}_f^\circ + \overline{h} - \overline{h}^\circ\right)_P = \sum N_R \left(\overline{h}_f^\circ + \overline{h} - \overline{h}^\circ\right)_R \longrightarrow \sum N_P \left(\overline{h}_f^\circ + \overline{h} - \overline{h}^\circ\right)_P = \sum N_R \overline{h}_{f,R}^\circ$$

since all the reactants are at the standard reference temperature of 25°C. Then, for the stoichiometric oxygen

$$C_2H_2 + 2.5O_2 \longrightarrow 2CO_2 + 1H_2O$$

From the tables,

,	$\overline{\mathbf{h}}_{\mathbf{f}}^{\circ}$	$\overline{\mathbf{h}}_{\mathbf{298K}}$	C <sub>2</sub> H <sub>2</sub> 25°C Combustion Products
Substance	kJ/kmol	kJ/kmol	$\begin{array}{c} 25^{\circ}\text{C} \\ \text{Combustion} \\ \text{chamber} \end{array}$
$C_2H_2(g)$	226,730		25°C
$O_2$	0	8682	
$N_2$	0	8669	
$H_2O(g)$	-241,820	9904	
$CO_2$	-393,520	9364	

Thus,

$$(2)\left(-393,520 + \overline{h}_{CO2} - 9364\right) + (1)\left(-241,820 + \overline{h}_{H2O} - 9904\right) = (1)\left(226,730\right) + 0 + 0$$

It yields 
$$2\bar{h}_{CO2} + 1\bar{h}_{H2O} = 1,284,220 \text{ kJ}$$

The adiabatic flame temperature is obtained from a trial and error solution. A first guess is obtained by dividing the right-hand side of the equation by the total number of moles, which yields 2,284,220/(2+1) = 428,074 kJ/kmol. The ideal gas tables do not list enthalpy values this high. Therefore, we cannot use the tables to estimate the adiabatic flame temperature In Table A-2b, the highest available value of specific heat is  $c_p = 1.234$  kJ/kg·K for CO<sub>2</sub> at 1000 K. The specific heat of water vapor is  $c_p = 1.8723$  kJ/kg·K (Table A-2a). Using these specific heat values,

$$(2)(-393,520 + \overline{c}_p \Delta T) + (1)(-241,820 + \overline{c}_p \Delta T) = (1)(226,730) + 0 + 0$$

where  $\Delta T = (T_{af} - 25)^{\circ}$ C. The specific heats on a molar base are

$$\overline{c}_{p,\text{CO2}} = \overline{c}_p M = (1.234 \text{ kJ/kg} \cdot \text{K})(44 \text{ kg/kmol}) = 54.3 \text{ kJ/kmol} \cdot \text{K}$$
  
 $\overline{c}_{p,\text{H2O}} = \overline{c}_p M = (1.8723 \text{ kJ/kg} \cdot \text{K})(18 \text{ kg/kmol}) = 33.7 \text{ kJ/kmol} \cdot \text{K}$ 

Substituting,

$$(2)(-393,520 + 54.3\Delta T) + (1)(-241,820 + 33.7\Delta T) = 226,730$$

$$(2 \times 54.3)\Delta T + 33.7\Delta T = 1,255,590 \text{ kJ/kmol}$$

$$\Delta T = \frac{1,255,590 \text{ kJ/kmol}}{(2 \times 54.3 + 33.7) \text{ kJ/kmol} \cdot \text{K}} = 8824 \text{ K}$$

Then the adiabatic flame temperature is estimated as

$$T_{\rm af} = \Delta T + 25 = 8824 + 25 = 8849$$
°C