

Force on vertical wall

Area of differential strip.
 $dA = w dy$

Pressure at y is same as at $(y+dy)$ as dy is very small.

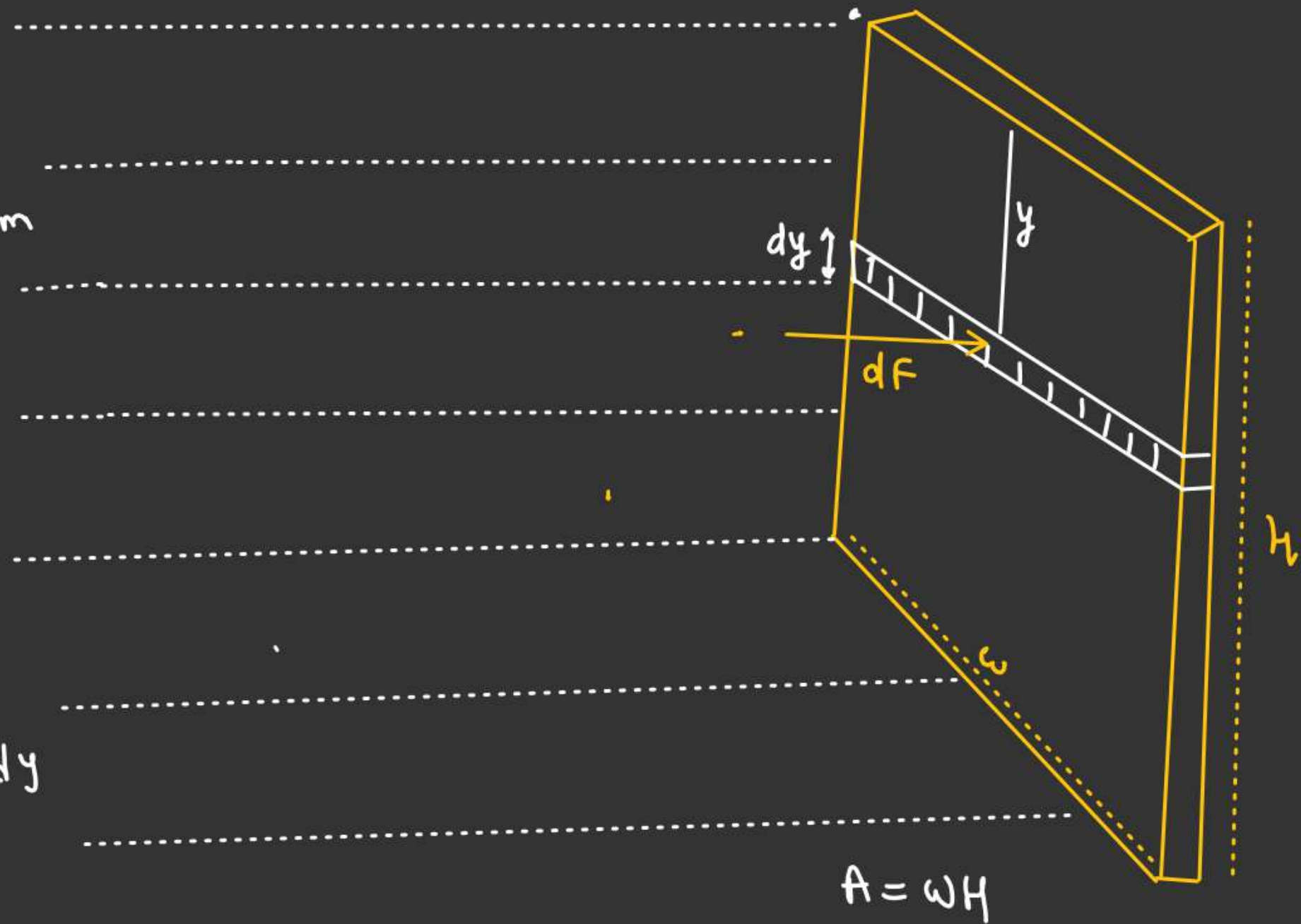
$$P_y = P_{y+dy} = \rho g y + P_{atm}$$

$$\int_0^H dF = \int_0^H P_y \cdot dA$$

$$= \int_0^H (P_{atm} + \rho g y) w dy$$

$$= P_{atm} w \int_0^H dy + \rho g w \int_0^H y dy$$

$$F = P_{atm} (\underline{wH}) + \frac{\rho g w H^2}{2}$$



Force on vertical wall

Area of differential strip.
 $dA = w dy$

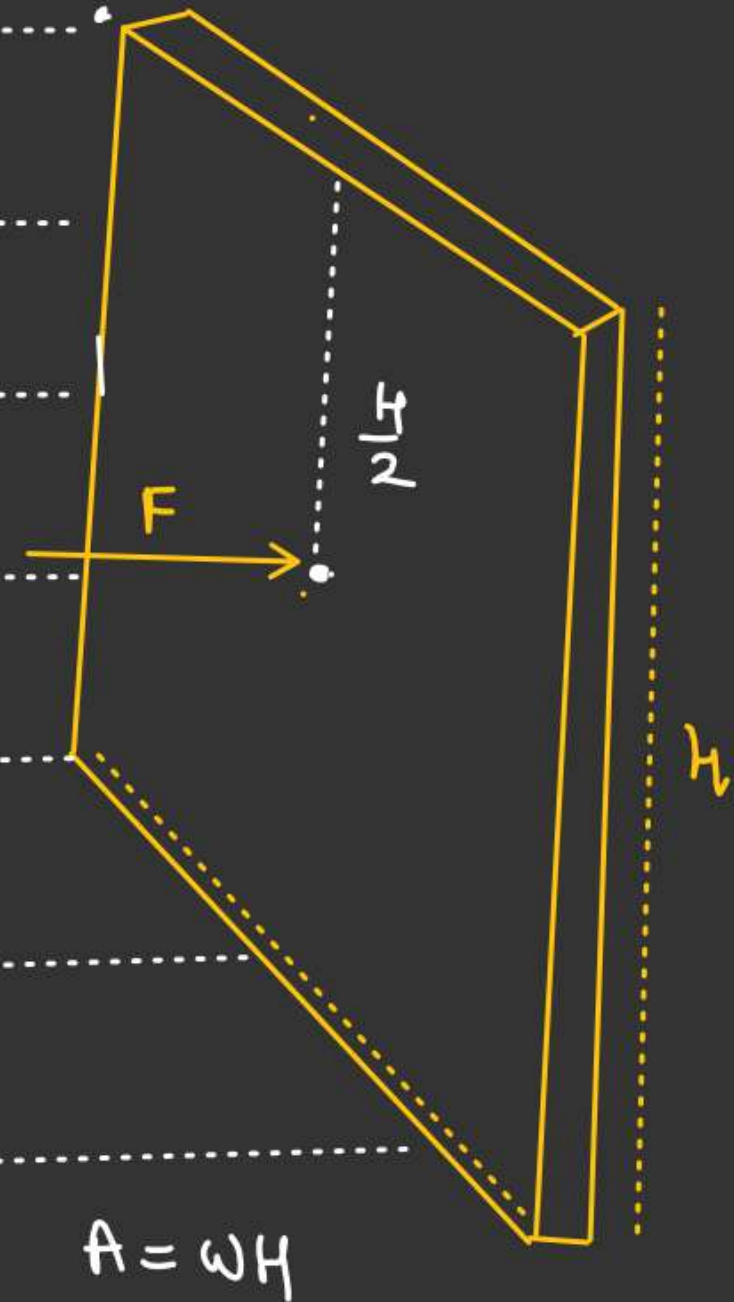
$$F = P_{atm}(wH) + \frac{\rho g w H^2}{2}$$

$$F = P_{atm} \cdot A + \left(\rho g \frac{H}{2} \right) (w \cdot H)$$

$$F = \underbrace{P_{atm} \cdot A}_{\downarrow} + \underbrace{\left(\text{Pressure at COM} \right)}_{\downarrow} \times \underbrace{\text{Area of plate}}_{\downarrow}$$

Due to
Atmospheric
pressure

Due to
Hydrostatic liquid.
i.e. thrust



Force on vertical wall

Due to liquid only.

$$dF = P_y \cdot dA$$

$$\int dF = \int \rho g y \cdot dA$$

$$\int dF = \rho g \int y \cdot dA$$

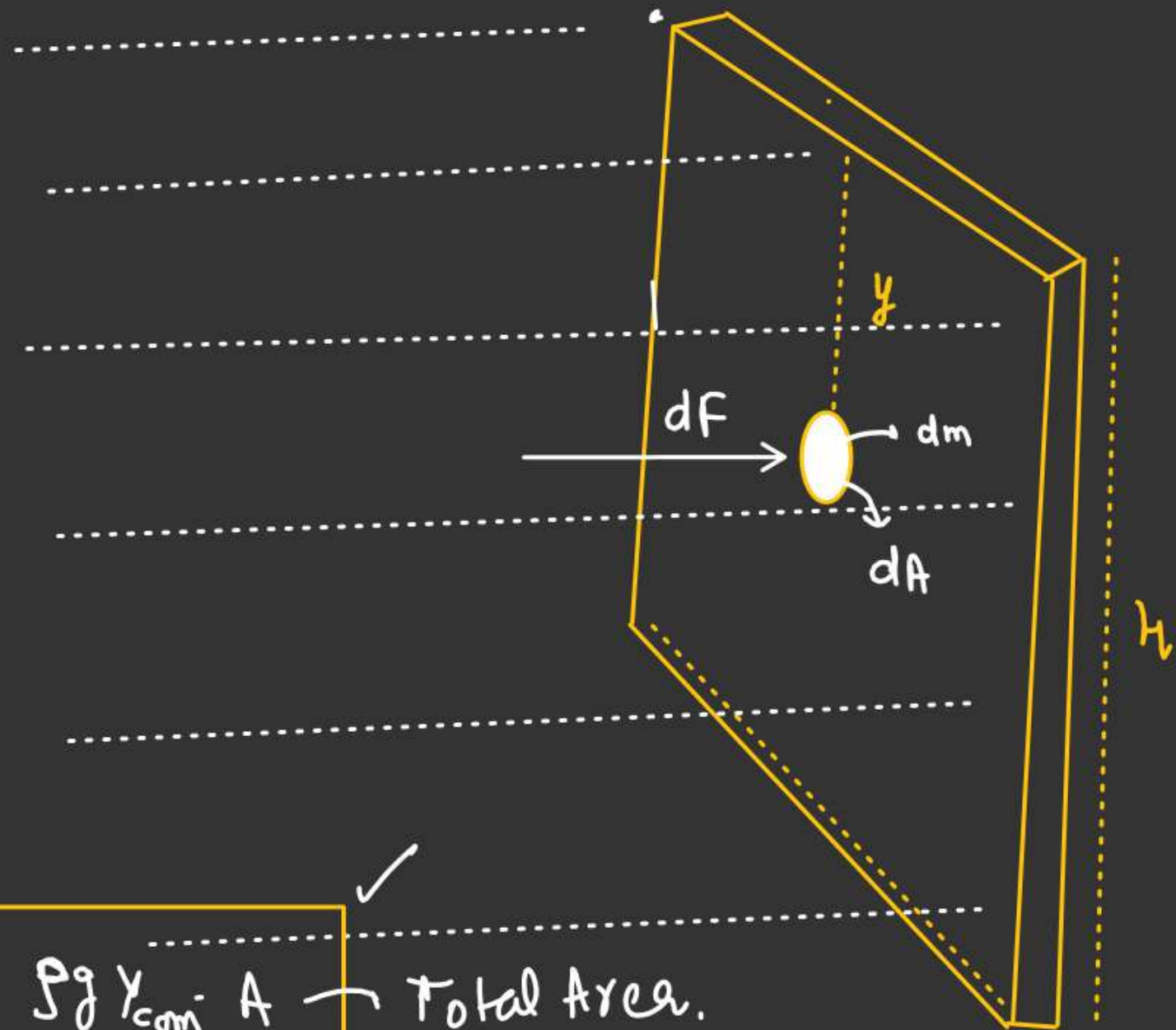
$$y_{com} = \frac{\int dm \cdot y}{\int dm}$$

$$dm = \sigma \cdot dA$$

$$y_{com} = \frac{\int y \cdot dA}{\int dA} = \frac{\int y \cdot dA}{A}$$

$$\underline{y_{com} \cdot A = \int y \cdot dA}$$

$$F_{net} = \underbrace{\rho g y_{com}}_{\text{Pressure at COM}} \cdot \underbrace{A}_{\text{Total Area}}$$



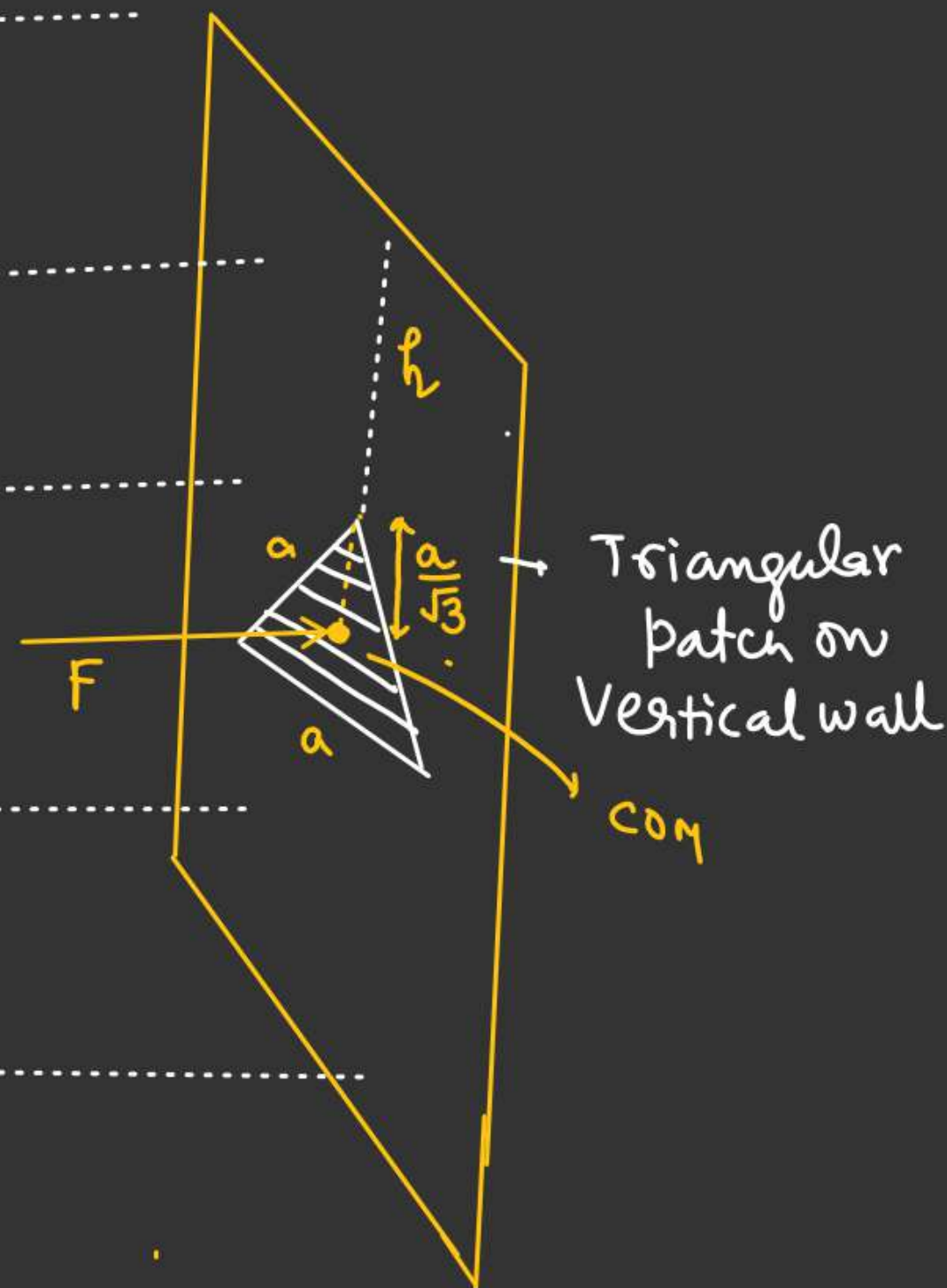
Hydrostatic thrust due to liquid on triangular patch

$$F_{\text{liquid}} = \underbrace{p g \left(h + \frac{a}{\sqrt{3}} \right)}_{\substack{\Downarrow \\ \text{Pressure at} \\ \text{COM.}}} \times \underbrace{\left(\frac{\sqrt{3} a^2}{4} \right)}_{\substack{\Downarrow \\ \text{Area of} \\ \text{triangle.}}}$$

$$F_{\text{atm}} = P_{\text{atm}} \cdot \left(\frac{\sqrt{3} a^2}{4} \right)$$

$$F_{\text{net}} = F_{\text{liquid}} + F_{\text{atm}}$$

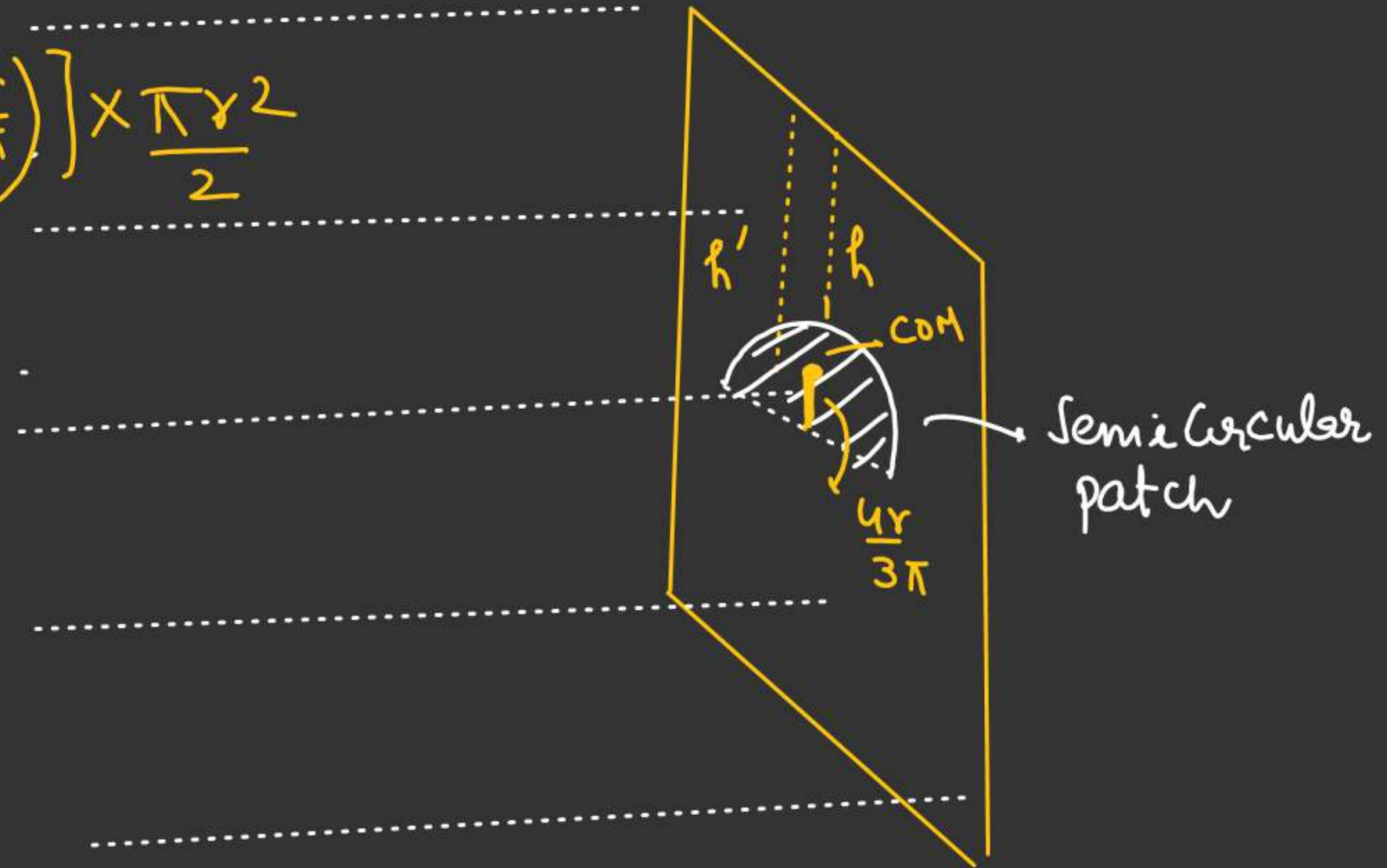
$$= P_{\text{atm}} \left(\frac{\sqrt{3} a^2}{4} \right) + p g \left(h + \frac{a}{\sqrt{3}} \right) \cdot \left(\frac{\sqrt{3} a^2}{4} \right)$$



QA

$$h' = h + \left(r - \frac{4r}{3\pi} \right)$$

$$F_{net} = P_{atm} \frac{\pi r^2}{2} + \rho g \left[h + \left(r - \frac{4r}{3\pi} \right) \right] \times \frac{\pi r^2}{2}$$



Find Hydrostatic thrust on the
Curve part of hemi Sphere.

$P_{atm} \rightarrow$ Neglected

M-1

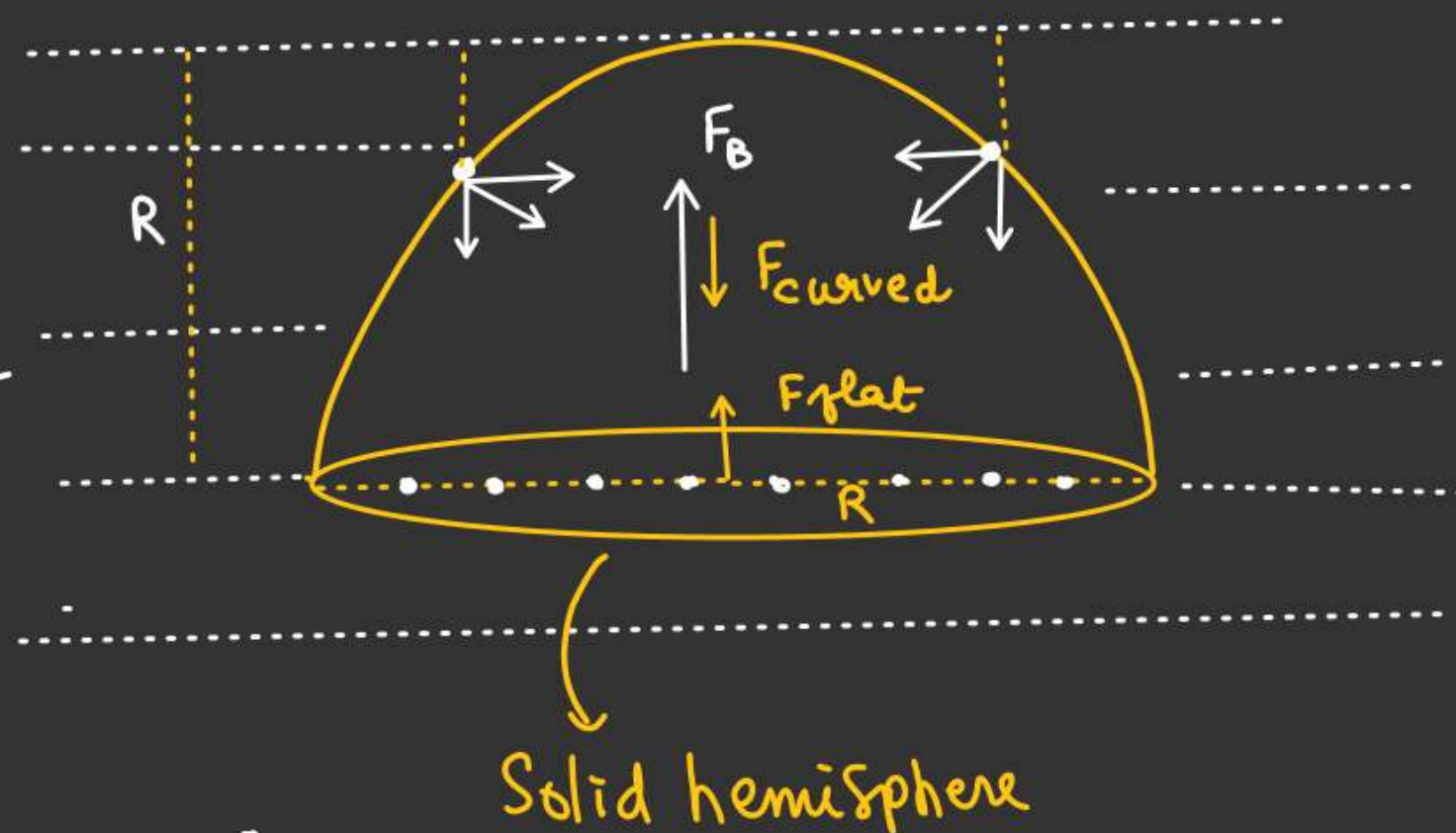
$$F_B = F_{flat\ part} - F_{curve\ part}$$

$$F_{curve\ part} = F_{flat\ part} - F_B$$

$$= \rho g R (\pi R^2) - \frac{2}{3} \pi R^3 \rho g$$

$$\therefore \rho g \pi R^3 - \frac{2}{3} \pi R^3 \rho g$$

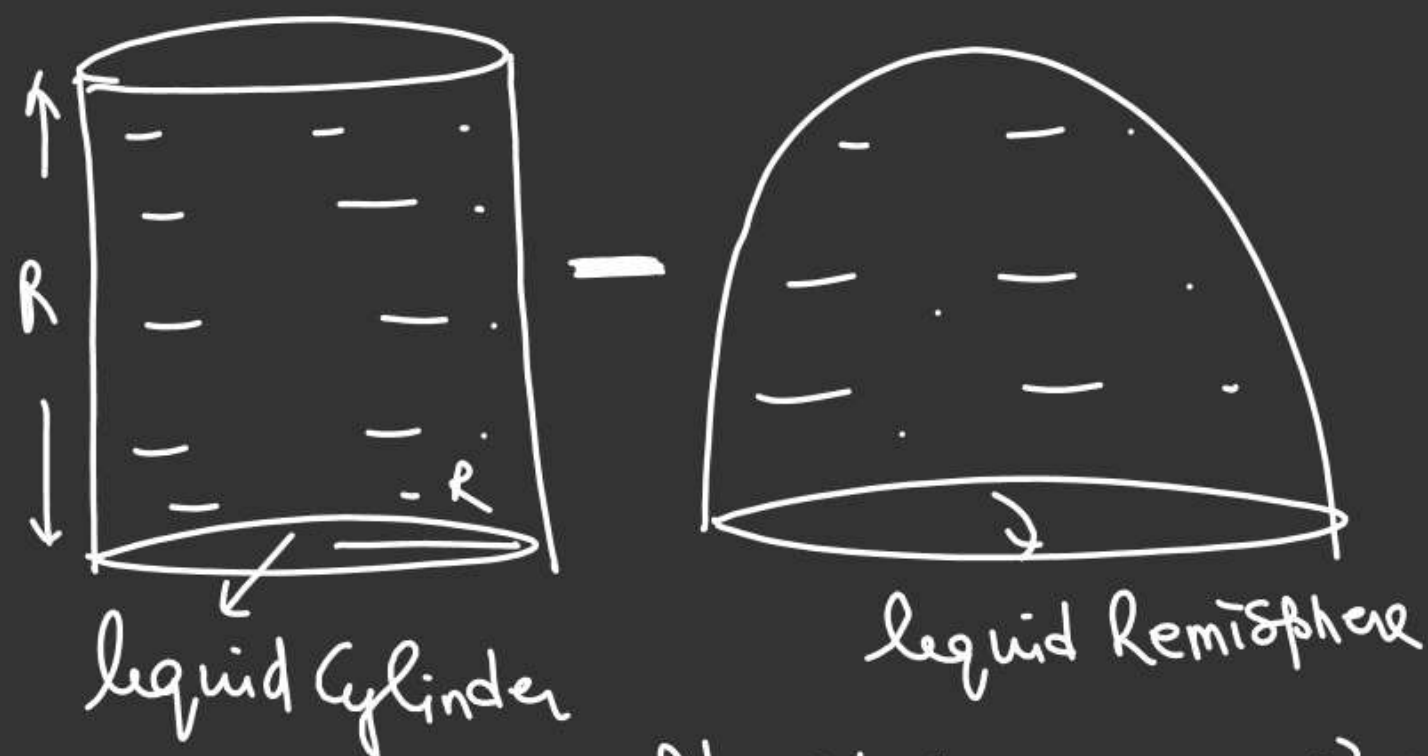
$$= \frac{1}{3} \rho g \pi R^3$$



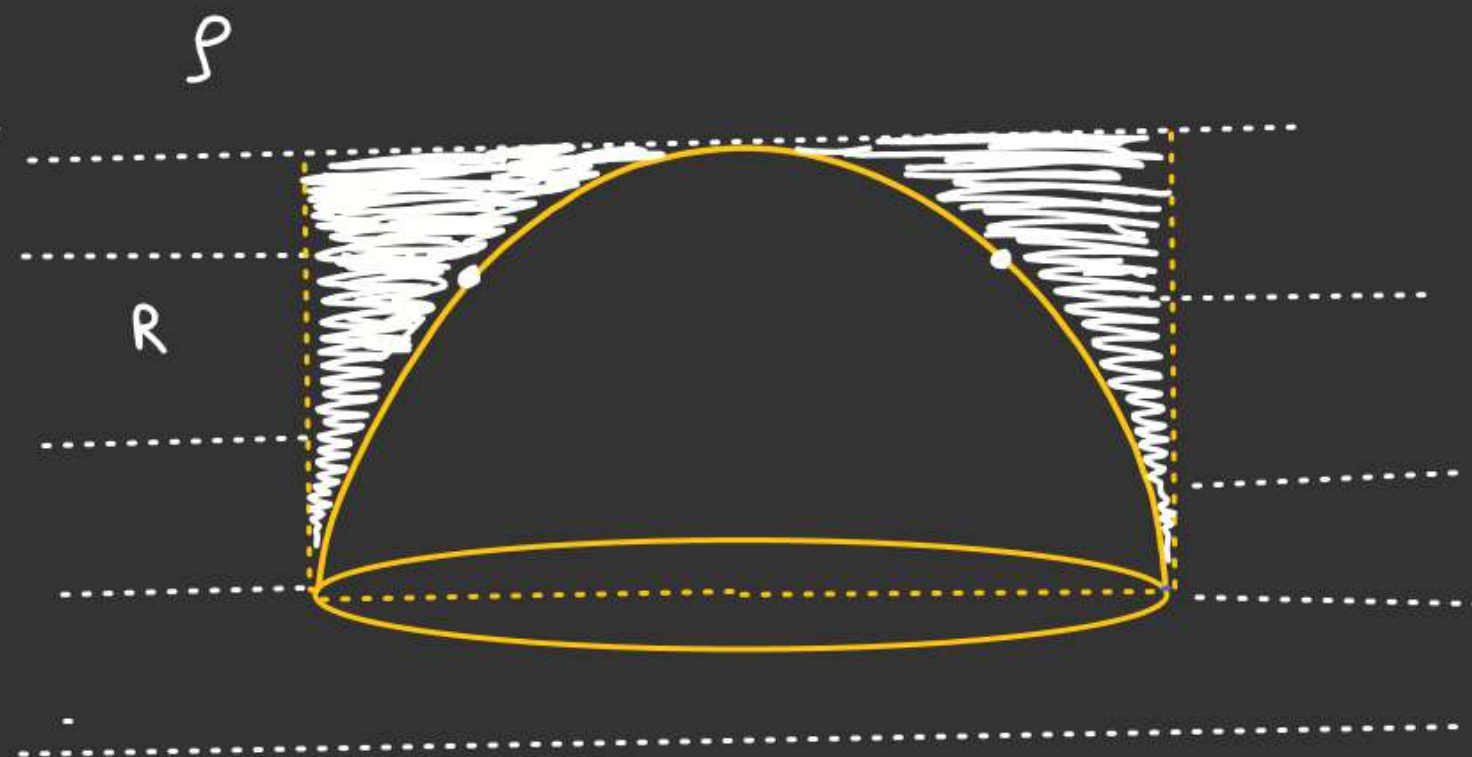
Find Hydrostatic thrust on the Curve part of hemi Sphere.

$P_{atm} \rightarrow$ Neglected

M-2:- Force on the Curve part is due to weight of the liquid just above the Curve part



$$\text{Volume of liquid above the curve surface} = \pi R^2 \cdot R - \frac{2}{3} \pi R^3 = \frac{1}{3} \pi R^3$$



$$\begin{aligned} F_{\text{curve part}} &= \text{Weight of liquid above curve part} \\ &= \left(\frac{1}{3} \pi R^3 \right) \rho \cdot g \\ &= \end{aligned}$$

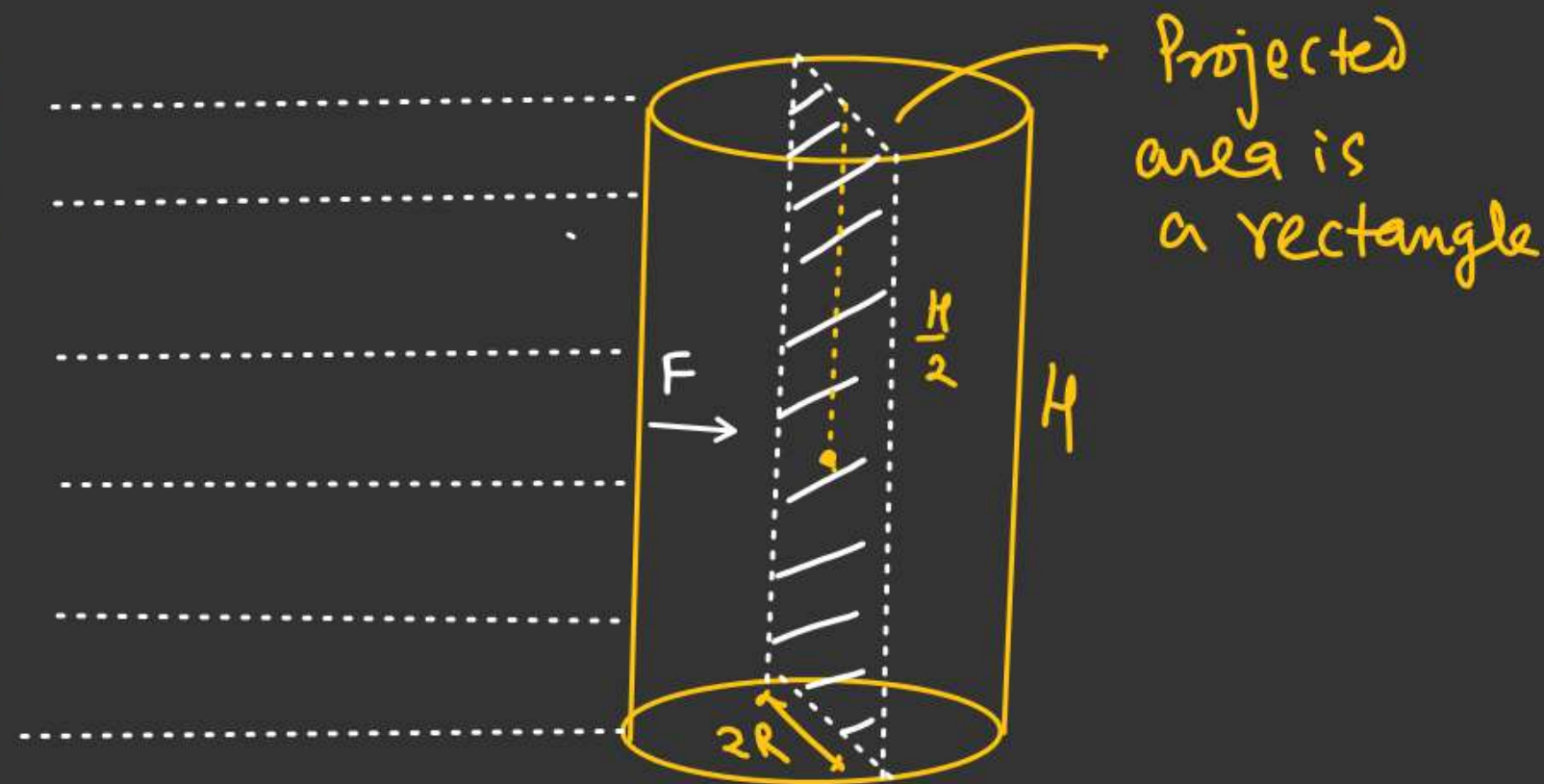
Nishant Jindal Find hydrostatic thrust on the cylinder.

$$F = \left(\rho g \frac{H}{2} \right) \left(\begin{array}{c} \text{Projected} \\ \text{area} \\ \perp \text{ to } F \end{array} \right)$$

$$F = \left(\rho g \frac{H}{2} \right) (\text{Area of Rectangle})$$

$$F = \left(\rho g \frac{H}{2} \right) (2R \cdot H)$$

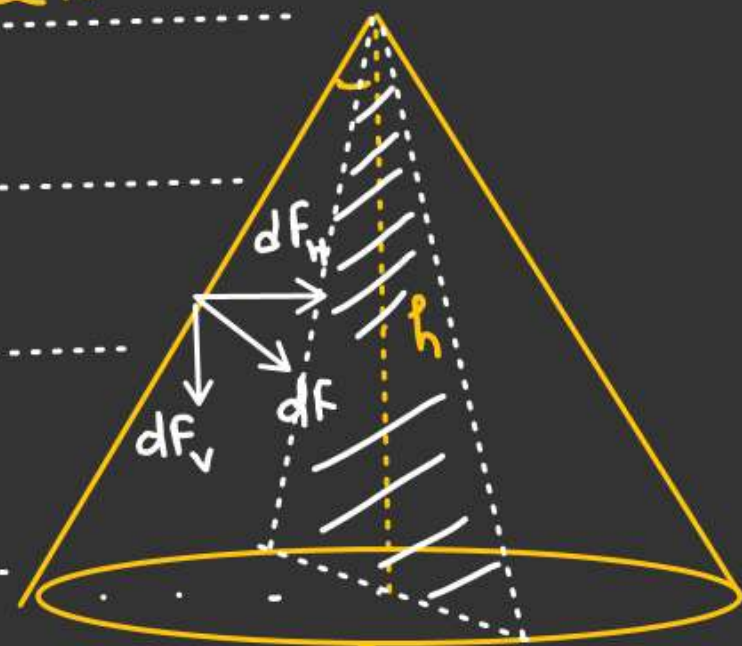
$$= \underline{\underline{\rho g R H^2}}$$



Find Force due to liquid on the cone.

H.W ✓ (Prove by integration)

$F_{\text{Horizontal}} = ?$, $F_{\text{vertical}} = ?$ ρ liquid



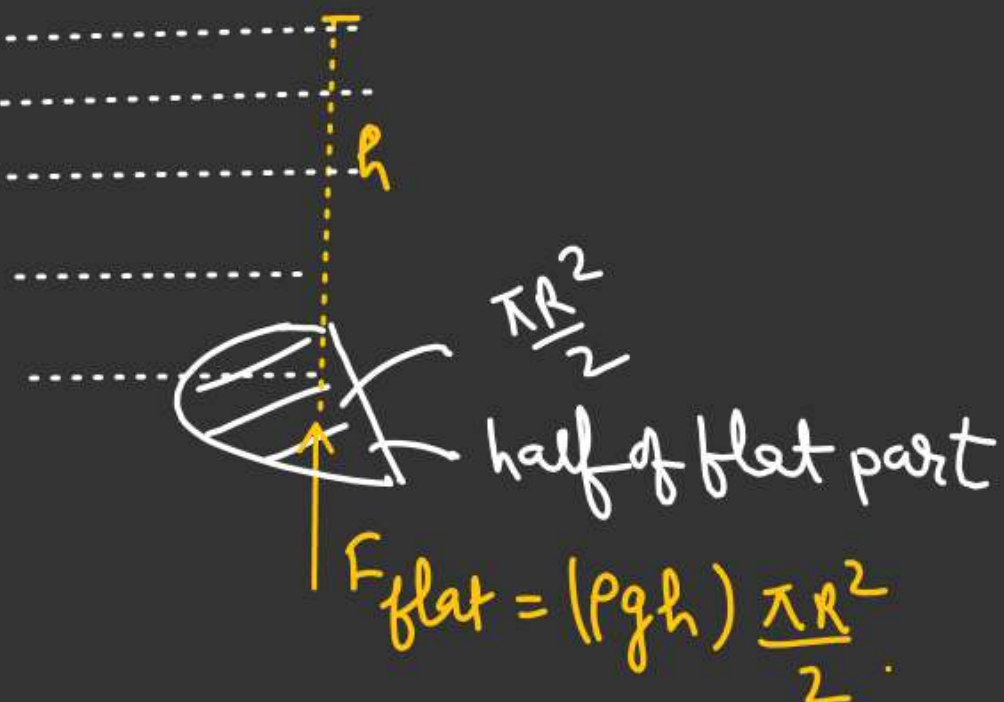
$$\underline{F_v} = F_{\text{flat part}} - F_B$$

$$= (pgh) \left(\frac{\pi R^2}{2} \right) - \left(\frac{1}{3} \pi R^2 h \right) \frac{1}{2} \rho g$$

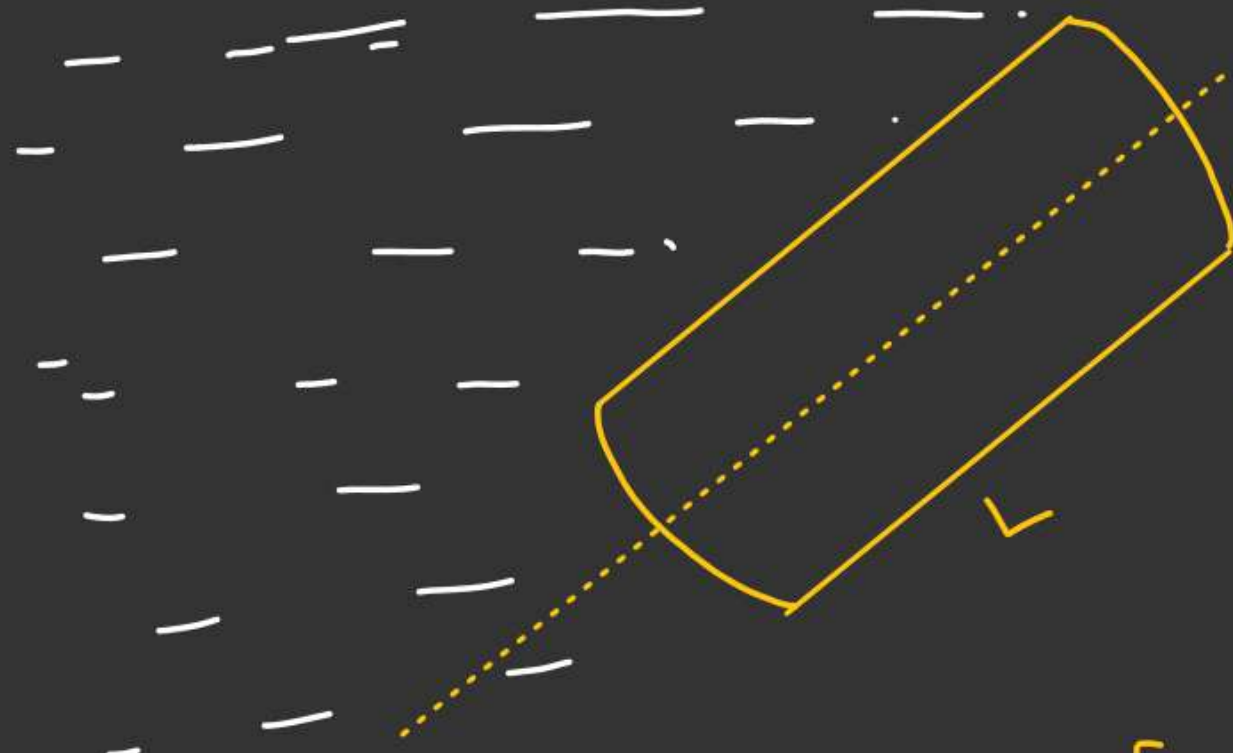
$$= \frac{pgh \pi R^2}{2} - \frac{\pi R^2 h \rho g}{6}$$

↓
volume
of half of
cone

$$= \left(\frac{pgh \pi R^2}{3} \right)$$



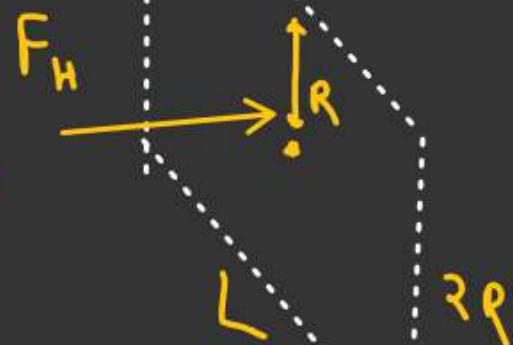
- 1) Find net hydrostatic torque on the dam ✓
- 2) Find hydrostatic force (Net)



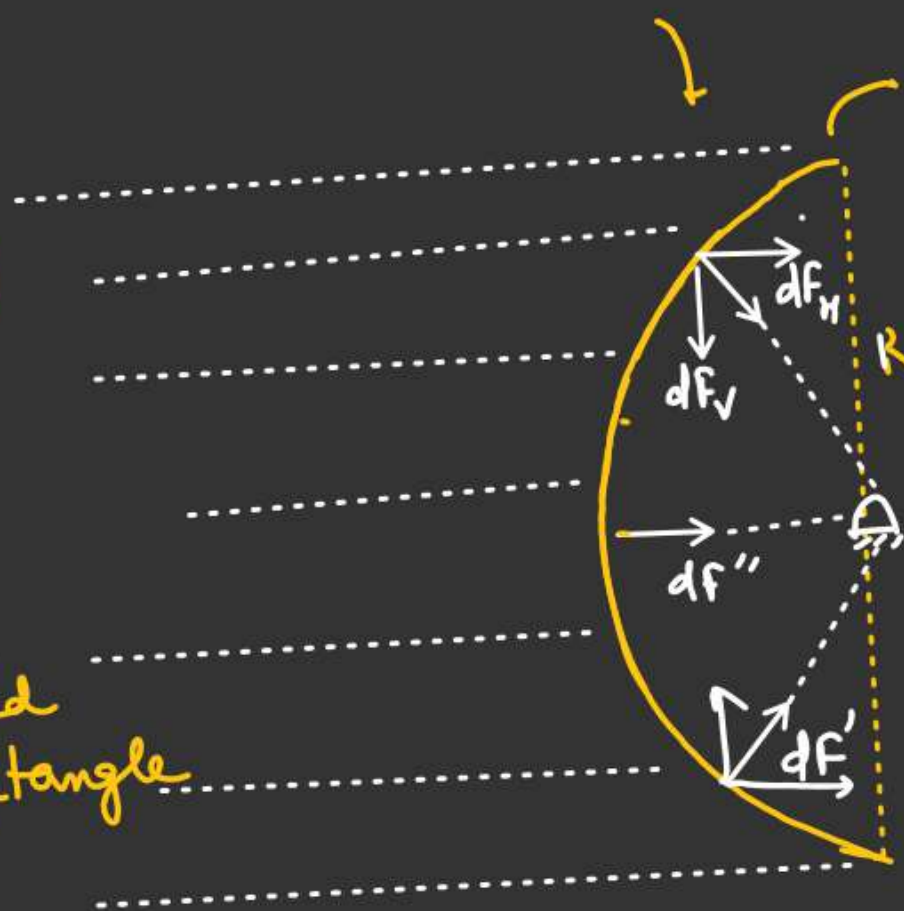
projected
Area Rectangle

$$F_H = (\rho g R)(L \times 2R)$$

$$F_H = \frac{2\rho g R^2 L}{2}$$



$$F_{net} = \sqrt{F_V^2 + F_H^2} \quad \checkmark$$



Cylindrical

Free to Rotate
length of Cylinder
is L.

Pressure force Normal to Curve
Surface so it passes through
hinge.
No net torque.

$$F_V = F_B = \left(\frac{\pi R^2 L}{2} \right) \rho g$$

$$= \frac{\rho g \pi R^2 L}{2}$$