ROB 498/599 Fall 2024: Assignment #6

Due on Dec 6th, at 11:59 pm, on Gradescope.

Submission Instructions: Submit a PDF file with written answers to all questions, images of plots (included on the PDF). Paste your code to the end of the document. Label the document [uniquename]_ROB498_Assignment6.pdf and upload to Gradescope. No late submissions will be accepted.

Problem 1 Consider the series elastic actuator (SEA) system we discussed in Assignment 4. Let the state variables be

$$x_1 = \theta_l$$
, $x_2 = \dot{\theta}_l$, $x_3 = \theta_m$, $x_4 = \dot{\theta}_m$,

and the state vector be $\mathbf{x} = \begin{bmatrix} x_1 & x_2 & x_3 & x_4 \end{bmatrix}^T$, then the state-space representation of this system is

$$\dot{\mathbf{x}} = A\mathbf{x} + Bu ,$$

where

$$A = \begin{bmatrix} 0 & 1 & 0 & 0 \\ -\frac{k}{J_l} & -\frac{B_l}{J_l} & \frac{k}{J_l} & 0 \\ 0 & 0 & 0 & 1 \\ \frac{k}{J_m} & 0 & -\frac{k}{J_m} & -\frac{B_m}{J_m} \end{bmatrix} , \quad B = \begin{bmatrix} 0 \\ 0 \\ 0 \\ \frac{1}{J_m} \end{bmatrix} .$$

(a) Assume we use a constant-gain, full-state feedback with the form

$$u = -\mathbf{K}\mathbf{x}$$
,

where $\mathbf{K} \in \mathbb{R}^4$. Write out the state-space representation of the closed-loop system.

(b) Use the same system parameters we used in Assignment 4, i.e., $J_m = J_l = 0.0097 \,\mathrm{kg} \cdot \mathrm{m}^2$, $b_m = b_l = 0.04169 \,\mathrm{Ns} \cdot \mathrm{m}^{-1}$, $k = 100 \,\mathrm{Nm} \cdot \mathrm{rad}^{-1}$. Let $Q = \mathrm{diag}([1, 0.1, 1, 0.1])$ be a 4×4 diagonal matrix, and R = 1. Find the constant feedback gain matrix K that minimizes the cost function

$$J = \int_0^\infty \left[\mathbf{x}^T(t) Q \mathbf{x}(t) + Ru^2(t) \right] dt . \tag{1}$$

- (c) Use the same system parameters. Let Q = diag([1, 0.1, 1, 0.1]) be a 4×4 diagonal matrix, and R = 0.1. Find the constant feedback gain K that minimizes the cost function (1).
- (d) Use the same system parameters. Let Q = diag([5, 0.1, 5, 0.1]) be a 4×4 diagonal matrix, and R = 0.1. Find the constant feedback gain K that minimizes the cost function (1).
- (e) Let the initial condition be $\mathbf{x}_0 = \begin{bmatrix} \pi/2 & 0 & \pi/2 & 0 \end{bmatrix}^T$. Simulate the closed-loop system response with the feedback gains you obtained in (b), (c), and (d) for 5 seconds. Plot the state variables \mathbf{x} , and control input u. Compare the results, what are the differences and why?

Problem 2 Consider a mass-spring-damper system described by the equation of motion

$$m\ddot{x} + b\dot{x} + kx = u$$
,

where m, b, k are the mass, damping coefficient, and spring stiffness. x represents the displacement from the origin, and u represents the external force. This can be written in the state-space form

$$\dot{\mathbf{x}} = A\mathbf{x} + Bu .$$

where **x** is the state vector. In this problem, use m = 1 kg, b = 2 Ns/m, k = 1 N/m.

(a) Suppose we have full-state measurement, and the external force u is determined by

$$u = -K\mathbf{x} , \qquad (2)$$

where $K = \begin{bmatrix} 5 & 1 \end{bmatrix}$. Suppose the initial condition is $x_0 = 1$, $\dot{x}_0 = 0$, simulate the closed-loop system response. Plot x and \dot{x} .

(b) The feedback controller used in (a) requires full-state feedback. However, in practice, full-state measurements may not be available. Suppose the only measurement you have is position, i.e.,

$$y = C\mathbf{x}$$
,

where $C = \begin{bmatrix} 1 & 0 \end{bmatrix}$. Construct a state observer with $L = \begin{bmatrix} 5 & -1 \end{bmatrix}^T$. Repeat simulation in (a), but replace \mathbf{x} in (2) with $\hat{\mathbf{x}}$, the estimated state from your state observer. Additionally, since you need to "guess" the initial condition, let $\hat{x}_0 = 1.2$ and $\dot{\hat{x}}_0 = 0.2$. Plot the results of your simulation.

(c) Construct another state observer, but this time let L be the gain computed using 1qe function in MATLAB. Let G = I be a 2×2 identity matrix, Q be a 2×2 diagonal matrix with diagonal entries $q_1 = q_2 = 10^{-4}$, and R = 0.1 (If you are wondering the reason we choose these values, check the next part). Simulate the system response with initial condition $\hat{x}_0 = 1.2$, $\dot{\hat{x}}_0 = 0.2$. Plot the result and compare with results of (b). Can you explain the difference?

Hint: Compare the eigenvalues of (A - LC).

(d) From (b) and (c), it seems that using L obtained from 1qe doesn't give the best results. However, in practice, measurement always comes with noise. Sensors always have additive white Gaussian noise caused by electron motion, with Gaussian distributed amplitude.

Assume we have a precise model of the system dynamics (this implies small process noise), but the position measurement has additive white Gaussian noise, with mean $\mu = 0$ and standard deviation $\sigma = 0.1$. The position measurement can be expressed as

$$\tilde{y} = y + z$$
,

where y is the ground-truth position, and $z \sim \mathcal{N}(\mu, \sigma^2)$ is the noise. Repeat (b) and (c) and compare the results. What are the differences and how can you explain it? You don't need to submit any plots.

Hint: You can generate a random number from a standard normal distribution $z \sim \mathcal{N}(0,1)$ using randn function in MATLAB.