# Chapitre V

# Etude des régimes compressibles en Fluide Parfait

→ Part 1 : Introduction

Description of compressible and inviscid flows Steady unidirectional compressible flows

→ Part 2 : From pressure waves to shock waves

Normal and oblique shock waves

Examples

Avertissement : Ce chapitre est tiré du cours de Master donné dans le cadre du « Summer Program » associant les écoles aéronautiques du GEA. Seule la première partie sera traitée.

#### Introduction (1)

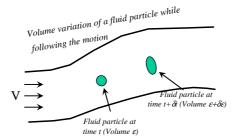
- Compressibility phenomena are associated with large velocities or large accelerations in a gas flow.
- The development of this field of fluid mechanics is linked with the evolution of aeronautics in the middle of the 20<sup>th</sup> century.
- → In fact, it concerns a lot of applications :
  - ⇒ Pneumatic transport
  - ⇒ flows in Intake or exhaust ports of automotive engines
  - ⇒ .....

#### Description of compressible and inviscid flows (1)

- Goals:
  - ⇒ Physical description
  - ⇒ Mathematical model
  - ⇒ Definition and properties of stagnation quantities

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#### Description of compressible and inviscid flows (2)



- What are the consequences of fluid motion on the variation of the volume of an elementary fluid particle?
- M = V/a➤ Important parameter : Mach Number
- First estimation (  $m = \rho \varepsilon$  constant mass of the fluid particle)

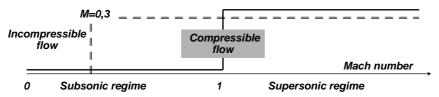
• 
$$\left| \frac{\delta \varepsilon}{\varepsilon} \right| = \left| \frac{\delta \rho}{\rho} \right| = \frac{1}{a^2} \left| \frac{\delta p}{\rho} \right|$$
  
• If  $\rho \approx \text{constant}$ :  $\delta p \approx \rho \frac{V_{\infty}^2}{2}$ 



$$\left| \frac{\delta \varepsilon}{\varepsilon} \right| \approx \frac{M_{\infty}^2}{2}$$

#### Description of compressible and inviscid flows (3)

--- Regimes of compressible flows



### Mach number : M=V/c (c: speed of sound)

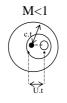
(For air in standard conditions of temperature and pressure, M=1 corresponds to 330 m/s)

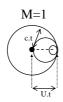
→ In some flows, for example on airfoils, both subsonic and supersonic regions can co-exist. We say that the flow regime is transsonic

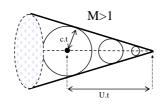
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#### Description of compressible and inviscid flows (4)

- → On both sides of Mach ONE!
- → SOURCE IN MOTION



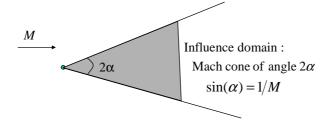




- $\implies$  Subsonic regime : Information arrives before the source
- ⇒ Supersonic regime : Information arrives after
  - Mach cone of angle  $\alpha / \sin(\alpha) = 1/M$
- ⇒ The properties of the equations of motion are changing :
  - \* Subsonic : System elliptic in space
  - \* Supersonic : System hyperbolic in space

#### Description of compressible and inviscid flows (5)

- → On both sides of Mach ONE
- → FIXED SOURCE in a Supersonic flow



- ⇒ In a supersonic flow, the fluid particle is not "informed" that there is an obstacle !!
  - This explains why we observe very sharp transitions
  - Explains the apparition of shock waves.

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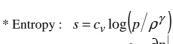
#### Mathematical model for compressible and inviscid flows. (1)

- → Large Reynolds numbers
- → Non buoyant fluid, No volumetric heat transfer
- → We assume that the fluid is a perfect gaz.
  - \* Equation of state :  $p/\rho = rT$  where r = R/M
  - ris the material specific gaz constant. \* Joule Laws:  $de = c_v dT$ ;  $dh = c_p dT$ r is the material specific heats

    or and c, are specific heats

    at constant pressure and volume  $c_v$  and  $c_p$  supposed constant;  $\gamma = c_p/c_v$

Meyer relation :  $r = c_p - c_v$ 



\* Velocity of sound :  $a^2 = \frac{\partial p}{\partial \rho} \bigg|_{s} = \gamma rT = \gamma \frac{p}{\rho}$ 

Tables and numerical integration must be used for more complex thermodynamics.

Mathematical model for compressible and inviscid flows. (2)

- → Mass, Momentum and energy balance are written .
- → If there are no irreversibilities and no volumetric heat flux, For a general unsteady flow :

$$\Rightarrow \frac{Ds}{Dt} = 0$$

In a compressible and inviscid flow, the entropy is constant along trajectories.

→ BEWARE: This is only true « pieces by pieces » if there are shock waves.

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Mathematical model for compressible and inviscid flows. (3)

- → Mass, Momentum and energy balance are written .
- → If there are no irreversibilities and no volumetric heat flux

  Case of a permanent flow (which will be considered afterwards)

$$\Rightarrow \vec{V}.g\vec{r}ad(s) = 0$$

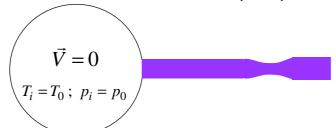
$$\vec{V}.g\vec{r}ad(h_i) = 0 \quad \text{where} \quad h_i = h + \frac{V^2}{2}$$

In a permanent compressible and inviscid flow the entropy and the stagnation enthalpy are constant along streamlines.

→ BEWARE: This is again only true « pieces by pieces » if there are shock waves.

## *Permanent regime: practical definitions of the stagnation quantities (1).*

 $\rightarrow$  Practical point of view : In a tank :  $h_i = c_p T_i = c_p T_0$ 

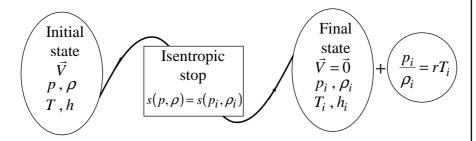


→ Following theoretical results, along streamlines :

\* 
$$h_i = h + \frac{V^2}{2}$$
 or  $T_i = T + \frac{V^2}{2c_p}$  are constant quantities

- → If then the properties of the gaz are uniform in the tank :
  - $\Rightarrow h_i$  et  $T_i$  are constant EVERYWHERE.
- Note that:  $h_i = \frac{a^2}{(\gamma 1)} + \frac{V^2}{2}$  and  $\frac{T_i}{T} = \left(1 + \frac{(\gamma 1)}{2}M^2\right)$

Permanent regime: practical definitions of the stagnation quantities (2).



- Following theoretical results, along streamlines:
  - \*  $h_i(p_i, \rho_i) = c_p T_i$  AND  $s(p_i, \rho_i)$  are independent constant quantities.
  - $\Rightarrow$   $T_i$ ;  $p_i$  et  $\rho_i$  are constant quantities along streamlines
- $\longrightarrow$  If then the properties of the gaz are uniform in the tank :
  - $\Rightarrow T_i$ ;  $p_i$  et  $\rho_i$  are constant EVERYWHERE. (Not valid if there are shock waves !!)

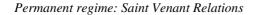
#### Permanent regime: Saint Venant Relations

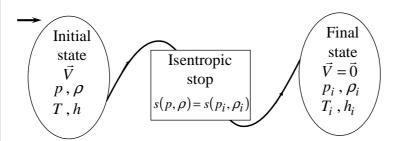
→ At a given Mach number M, the ratio between a quantity and the corresponding stagnation quantity are given by :

### **SAINT-VENANT RELATIONS**

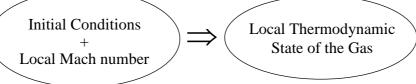
$$\begin{split} \frac{T}{T_i} &= \left(1 + \frac{\gamma - 1}{2} M^2\right)^{-1} & \frac{p}{p_i} = \left(1 + \frac{\gamma - 1}{2} M^2\right)^{-\gamma/(\gamma - 1)} \\ \frac{\rho}{\rho_i} &= \left(1 + \frac{\gamma - 1}{2} M^2\right)^{-1/(\gamma - 1)} & \frac{a}{a_i} = \left(1 + \frac{\gamma - 1}{2} M^2\right)^{-1/2} \end{split}$$

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→ In a permanent compressible and inviscid flow If the boundary conditions are UNIFORM, and if there are NO shock waves :

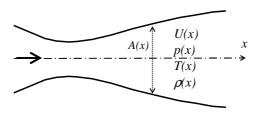


# Permanent compressible and inviscid Monodimensional flow

## Laval Nozzle Flow

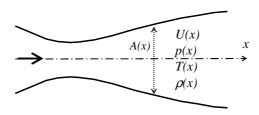
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Permanent compressible and inviscid monodimensional flow: Laval nozzle flow (1)



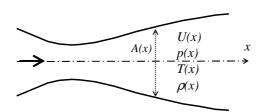
- \* Quasi-monodimensional flow along the x coordinate
- \* Slow variations of A(x) with dA/A << 1
- \* Weak curvatures  $A/R^2 \ll 1$
- \* Uniform boundary conditions

Permanent compressible and inviscid monodimensional flow: Laval nozzle flow (2)



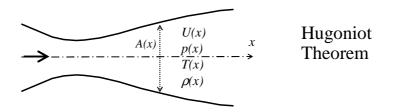
- \* Continuity :  $\rho.U.A = cste$
- \* Momentum :  $\rho.U.\frac{dU}{dx} = -\frac{dp}{dx}$
- \* Enthalpy :  $h_i = cste$
- \* Entropy :  $s = cste \Rightarrow \frac{p}{\rho^{\gamma}} = cste$

Permanent compressible and inviscid monodimensional flow: Laval nozzle flow (3)



- \* Continuity :  $\frac{d\rho}{\rho} + \frac{dU}{U} + \frac{dA}{A} = 0$
- \* Momentum :  $\rho.U.\frac{dU}{dx} + \frac{dp}{dx} = 0$
- \* Enthalpy :  $h_i = cste$
- \* Entropy :  $\frac{dp}{p} = \gamma \frac{d\rho}{\rho} \Rightarrow dp = a^2 d\rho$

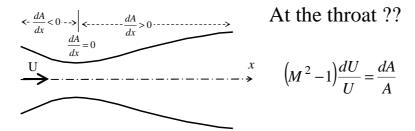
Permanent compressible and inviscid monodimensional flow: Laval nozzle flow (4)



- \* Reducing variables :  $(M^2 1)\frac{dU}{U} = \frac{dA}{A}$
- \* In a SUBSONIC FLOW, When section increases, The velocity decreases (and vice versa)
- \* In a SUPERSONIC FLOW, When section increases, The velocity increases (and vice versa)

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## Laval nozzle flow (5)

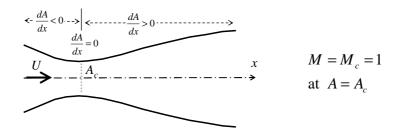


\* If M=1, then dA=0

If an isentropic monodimensional flow is sonic, then we are at a minimum of the cross-section.

- \* Conversely, if dA=0:
- $\Rightarrow$  Either dU=0
- $\Rightarrow$  Either M=1. In this case, we are at a MINIMUM of the section

#### Laval nozzle flow(6)



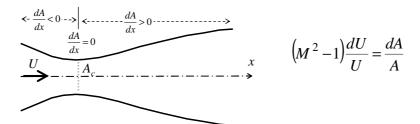
\* With a sonic flow at the throat, we obtain for the mass flux :

$$\Rightarrow Q_m = \sqrt{\frac{\gamma}{r}} \left(\frac{\gamma+1}{2}\right)^{-\frac{\gamma+1}{2(\gamma-1)}} \frac{p_i}{\sqrt{T_i}} A_c = \left(4.04.10^{-2}\right) \frac{A_c p_i}{\sqrt{T_i}} \qquad (\gamma = 1.4)$$

\* This relation has a lot of practical applications if one wants to regulate a mass flux just by controlling the initial stagnation pressure  $p_i$ 

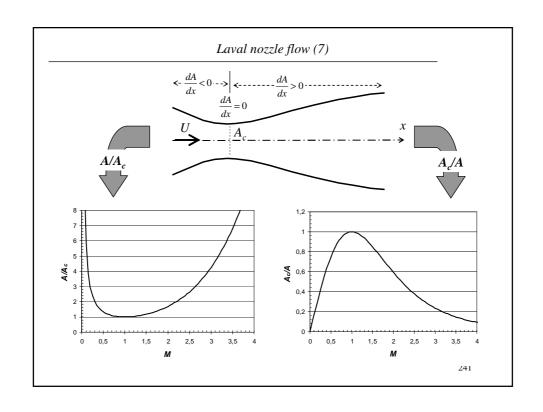
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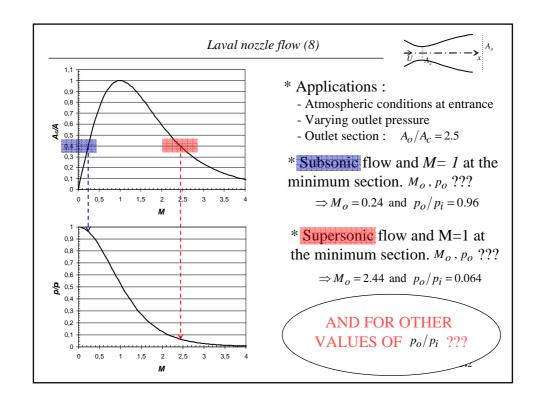
### Laval nozzle flow (7)

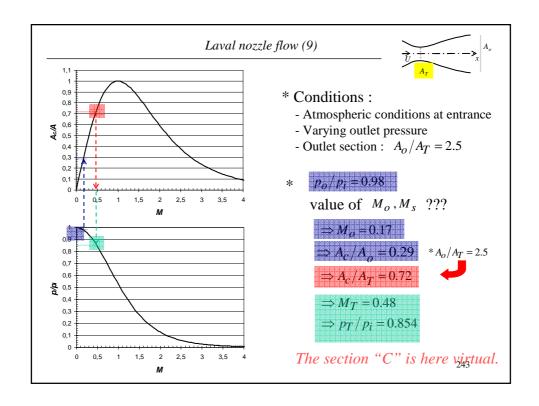


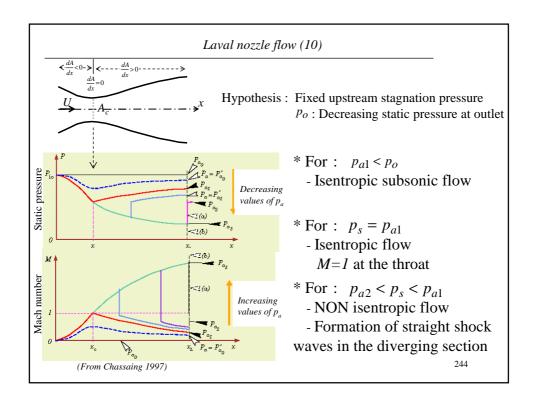
- \* By definition, the throat conditions are defined by :  $M = M_C = 1$  at  $A = A_C$  (This section can be virtual!!)
- \* An important theoretical link between  $A/A_c$  and the local Mach number is :

$$\Rightarrow \frac{A}{A_c} = \frac{1}{M} \left[ \frac{2}{\gamma + 1} \left( 1 + \frac{\gamma - 1}{2} M^2 \right) \right]^{\frac{\gamma + 1}{2(\gamma - 1)}}$$

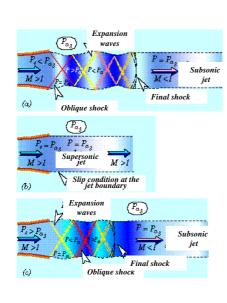








## Laval nozzle flow (11)



- \* For :  $p_{a3} < p_s < p_{a2}$
- Compression at the oulet by oblique shock waves
- \* For :  $p_s = p_{a3}$  Isentropic Supersonic flow
- \* For :  $p_s < p_{a3}$
- Expansion wave at the outlet reflecting on the boundary of the jet (p is a constant on this surface)

(From Chassaing 1997)