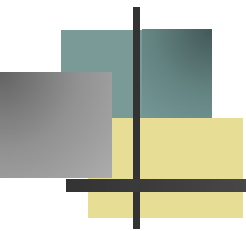




Chapter 8

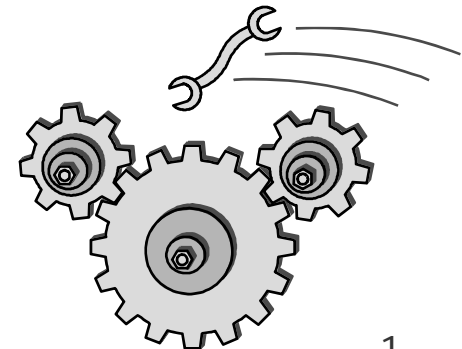
Design for Reliability



8.1 Reliability Specification

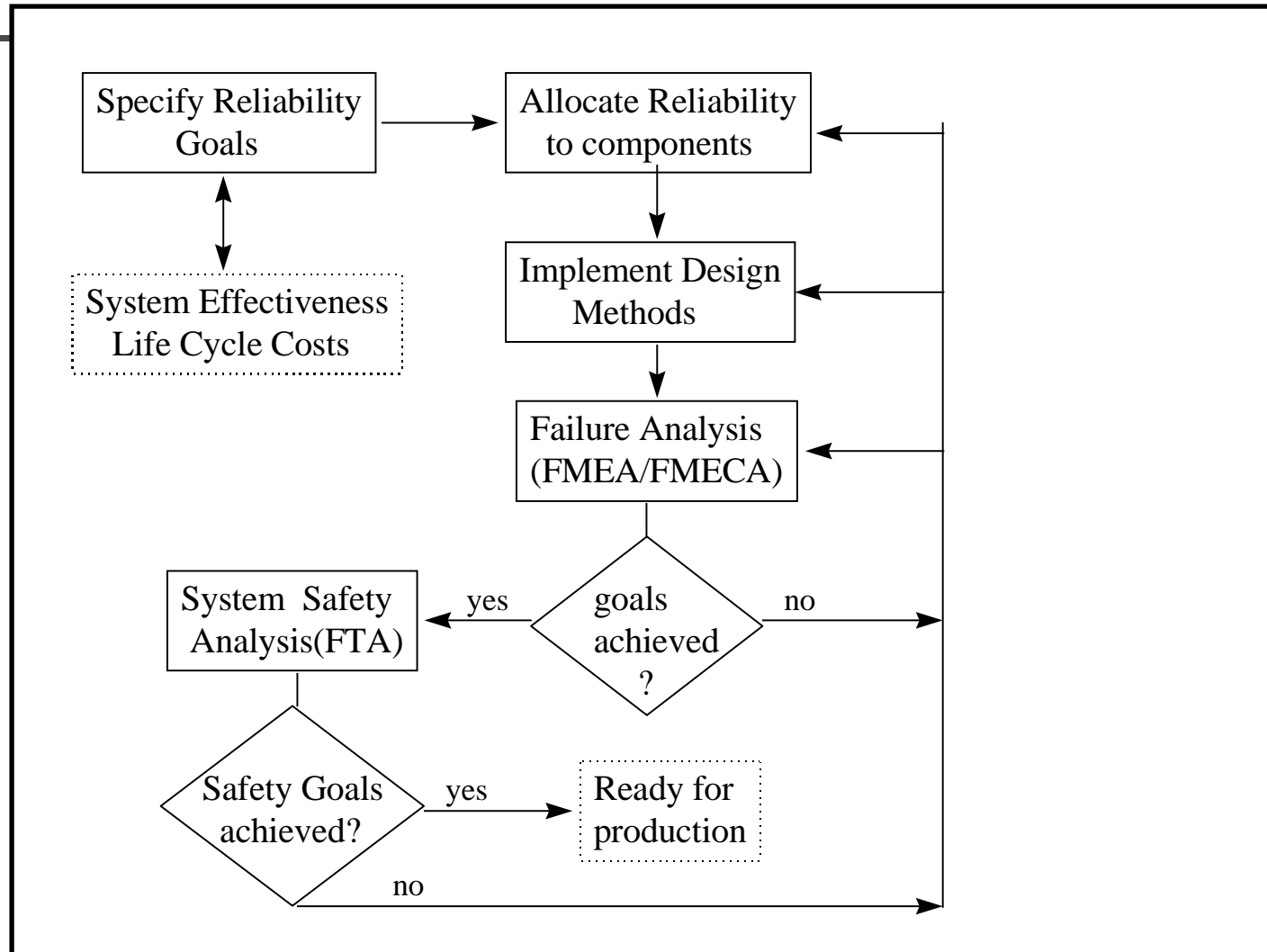
8.2 Reliability Allocation

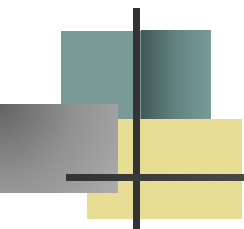
8.3 Design Methods



8.1 Reliability Specification

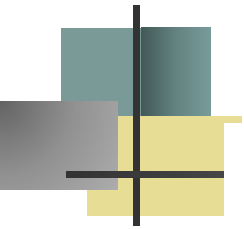
The Reliability Design Process



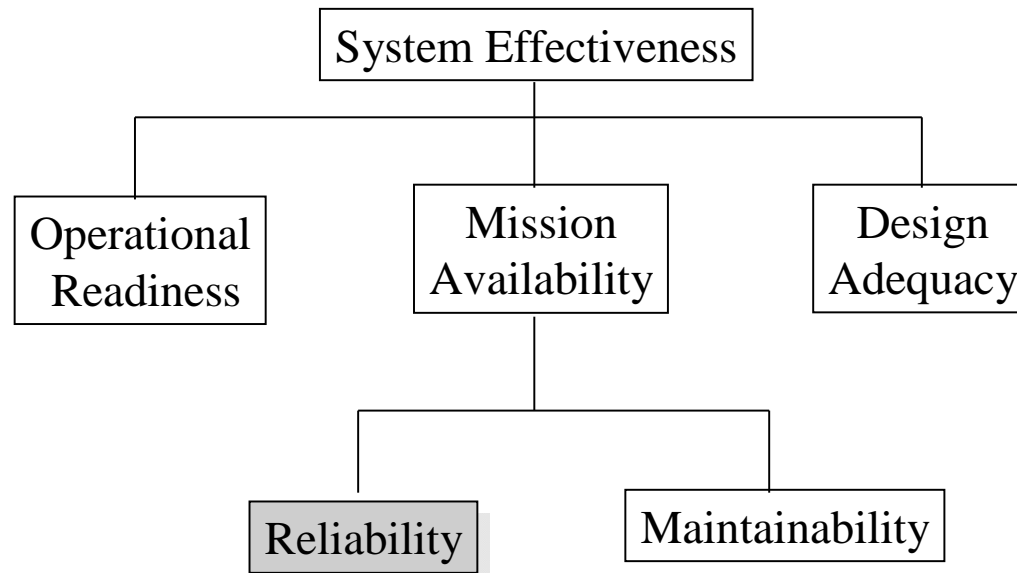


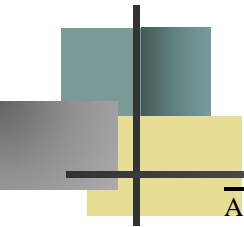
Reliability Activities & Product Life Cycle

Development Phase	Conceptual & Preliminary Design	Detailed Design, Development & Prototyping	Production & Manufacture	Product Use & Support
	Specification Allocation Design Methods	Design Methods Failure Analysis Growth Testing Safety Analysis	Acceptance test Quality Control Burn-in & Screen Testing	Preventive & Predictive Maintenance Modifications Parts Replacement



System Effectiveness





Life Cycle Cost Categories

Acquisition Costs	Operations and Support Costs	Phase-out
Research and Development Management Engineering Design and Prototyping Engineering Design Fabrication Testing & evaluation Production Manufacturing Plant facilities & overhead Marketing & Distribution	Operations Facilities Operators Consumables (energy & fuel) Unavailable time or downtime Support Repair resources Supply resources repairables expendables tools, test & spt equip Failure Costs Training Technical Data	Salvage value Disposal Costs





Life Cycle Cost

$$\text{LCC} = \text{Acquisition Costs} + \text{Operations Costs} + \text{Failure Cost} + \text{Support Costs} - \text{Net Salvage Value}$$

where $\text{Net Salvage Value} = \text{Salvage Value} - \text{Disposal Cost}$

Discount Monetary Values





8.2 Reliability Allocation

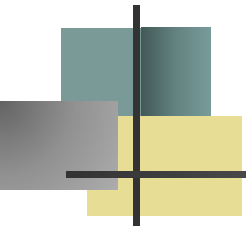
In general: $h(R_1(t), R_2(t), \dots, R_n(t)) \geq R^*(t)$

where $R_i(t)$ is the reliability at time t of the i th component,

and $R^*(t)$ is the system reliability goal at time t .

or $g(\text{MTTF}_1, \text{MTTF}_2, \dots, \text{MTTF}_n) \geq \text{MTTF}^*$

For series related components:
$$\prod_{i=1}^n R_i(t) \geq R^*(t)$$



Exponential Case

$$\prod_{i=1}^n e^{-\lambda_i t} \geq R * (t)$$

$$\sum_{i=1}^n \lambda_i \leq \lambda_s$$



ARINC Method

Assume components are in series, are independent, and have constant failure rates.

$$\text{new } \lambda_i = w_i \lambda^*$$

$$w_i = \frac{\lambda_i}{\sum_{i=1}^n \lambda_i} \quad i = 1, 2, \dots, n ; \text{ since}$$

$$\sum_{i=1}^n \text{new } \lambda_i = \sum_{i=1}^n w_i \lambda^* = \sum_{i=1}^n \frac{\lambda_i \lambda^*}{\sum_{i=1}^n \lambda_i} = \frac{\lambda^*}{\sum_{i=1}^n \lambda_i} \sum_{i=1}^n \lambda_i = \lambda^*$$



AGREE Method

t = system operating time

$R^*(t)$ = system reliability goal at time t

n = number of components

n_i = the number of modules within component i

N = total number of modules in system = $\sum n_i$

t_i = the operating time of the i th component, $t_i \leq t$

λ_i = the failure rate of the i th component

w_i = the probability the system will fail given component i has failed

Allocating an equal share of the reliability to each module results in component i 's contribution to the system reliability being

$$[R^*(t)]^{\frac{n_i}{N}}$$



AGREE Method

$$w_i (1 - e^{-\lambda_i t_i}) = 1 - [R^*(t)]^{\frac{n_i}{N}}$$

$$\lambda_i = -\frac{1}{t_i} \ln\left(1 - \frac{1 - R^*(t)^{n_i/N}}{w_i}\right)$$

note that $\prod_{i=1}^n e^{-\lambda_i t_i} \leq R^*(t)$



AGREE Method - Example

Component	Import Index (w_i)	Oper hrs (t_i)	Nbr of modules - n_i
Receiver	.8	1000	25
Antenna	1	1000	15
Transmitter	.7	500	23
Power Supply	1	1000	70

The total module count is 133. If the system reliability goal is .99, then the reliability to be allocated to the i th component is $.99^{n_i/133}$.

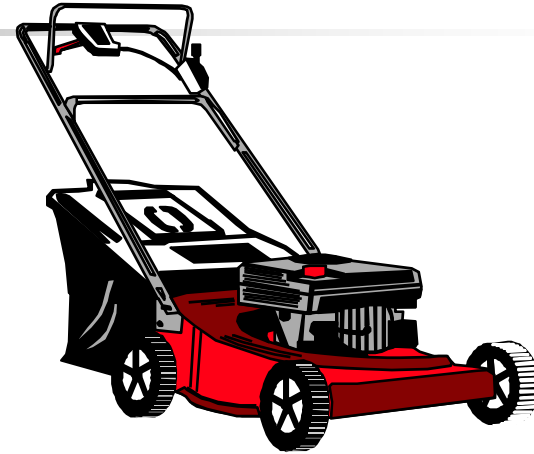
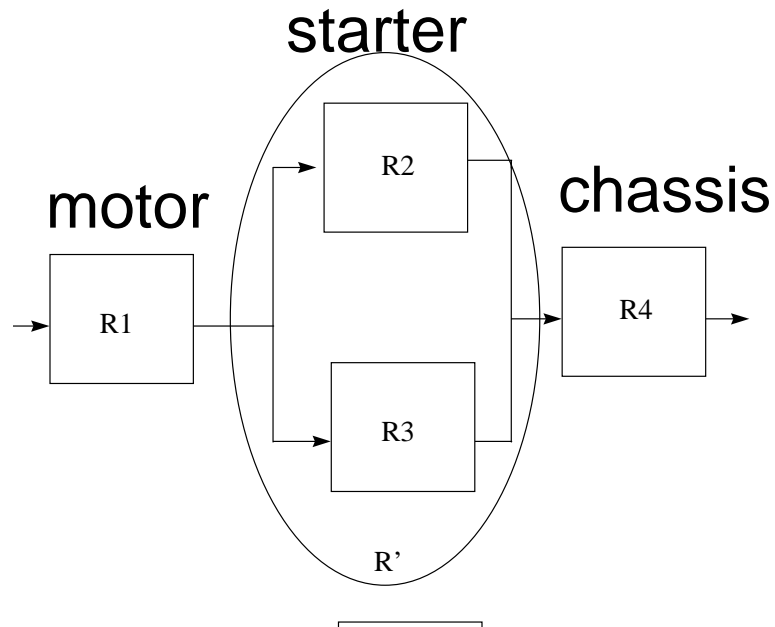


AGREE Method - Example

<u>Component</u>	<u>Failure Rate</u>	<u>MTTF</u>	<u>Reliability</u>	<u>System Rel</u>
Receiver	2.362×10^{-6}	423,369	.99764	.99811
Antenna	1.1335×10^{-6}	882,227	.99887	.99887
Transmitter	4.9676×10^{-6}	201,303	.99752	.99826
Power Supply	5.2896×10^{-6}	189,048	.99472	.99472
System	1.3753×10^{-5}	72,713	.98879	.99

From the above table, the probability of a component failure is $1 - .98879$ while the probability of a system failure is $1 - .99$.

Redundancy



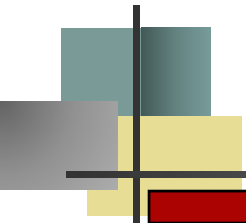
$$R^* = R1 \times R' \times R4$$

$$R' = 1 - (1 - R2) (1 - R3) = R2 + R3 - R2 R3$$

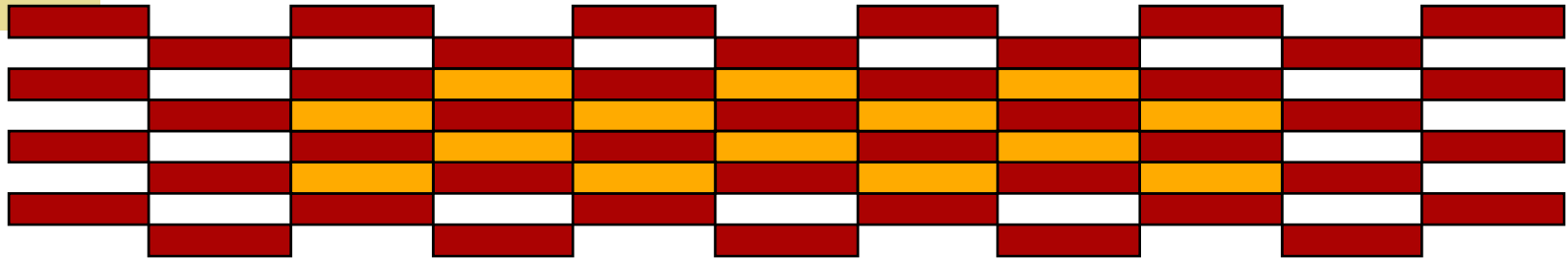
$$R_2 = \frac{R' - R_3}{1 - R_3}$$

$$\text{or } R' = 2R - R^2$$

$$\text{with solution } R = 1 - (1 - R')^{.5}$$



8.3 Design Methods



Parts and Material Selection

Derating

Stress-Strength Analysis

Complexity

Choice of Technology

Redundancy



Material Selection

Structure

atomic bonding
crystal structure
defect structure
microstructure

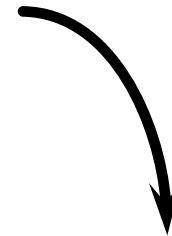
MATERIALS
SCIENCE



Material Properties

yield strength
hardness
fatigue life
creep

MATERIALS
ENGINEERING



Service Performance

Stresses
corrosion
temperature
radiation
vibration

MANUFACTURING
ENGINEERING



Manufacturing Process

casting
machining (cutting)
joining
heat treatment
assembly

$R(t)$



Design Methods

Parts and Material Selection

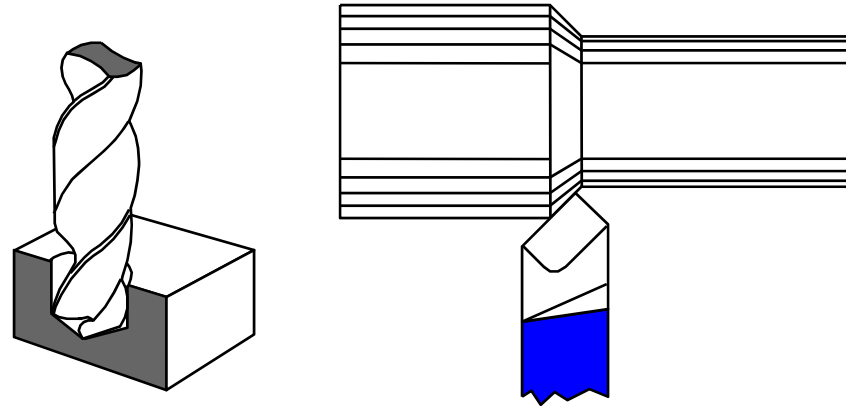
Tensile Strength

Hardness

Impact Value

Fatigue Life

Creep



Property of Materials

Metals

Composites

Polymers

Ceramics



Material Selection

Metals	Ceramics	Polymers
strong	strong	weak
stiff	stiff	compliant
tough	brittle	durable
electrically conducting	electrically insulating	electrically insulating
high thermal conductivity	low thermal conductivity	temperature sensitive



Tensile strength

Tensile strength measures the force required to pull something such as rope, wire, or a structural beam to the point where it breaks.

Specifically, the maximum amount of stress that it can be subjected to before failure

Yield strength - The stress a material can withstand without permanent deformation.

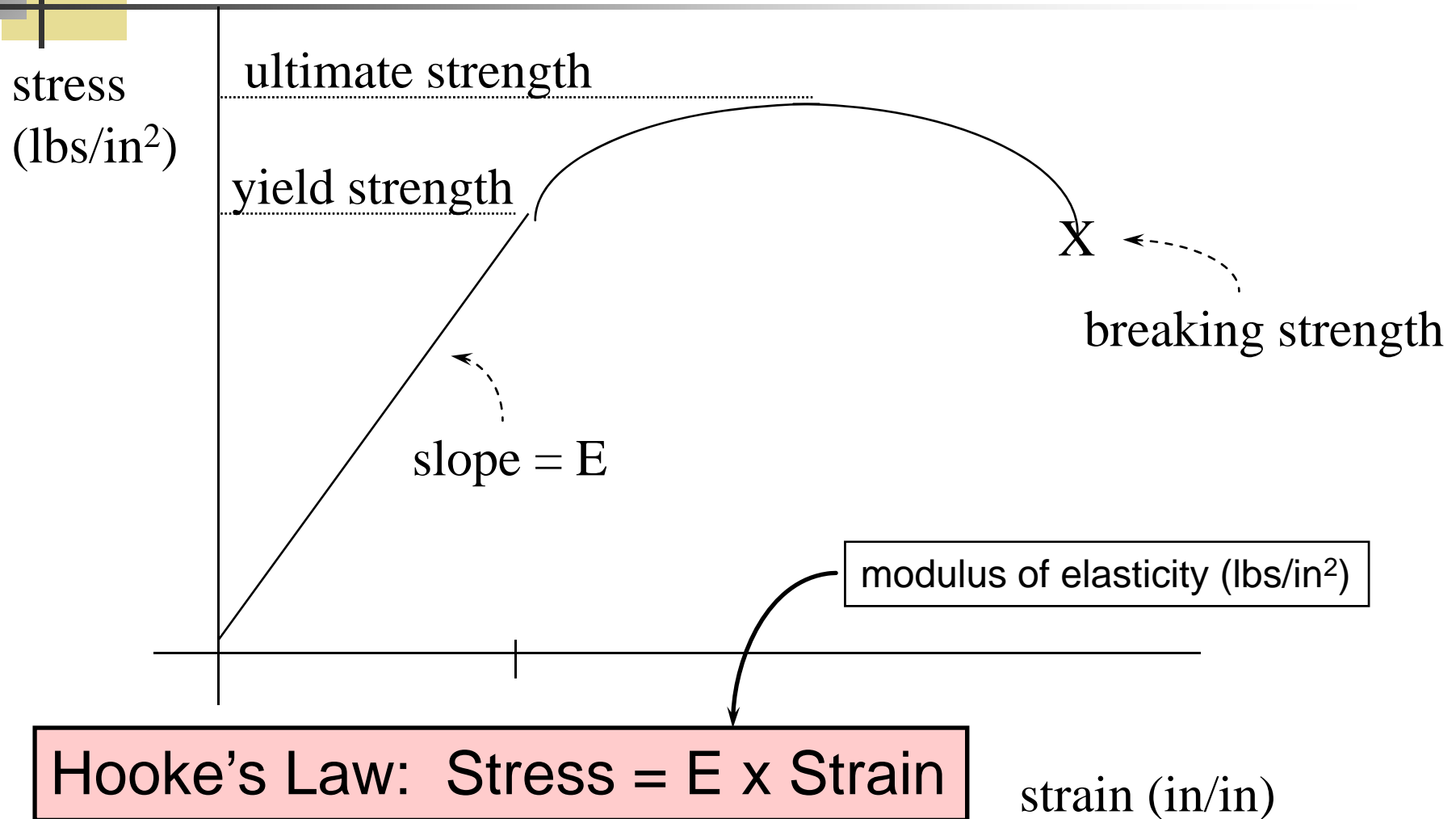
Ultimate strength - The maximum stress a material can withstand.

Breaking strength - The stress coordinate on the stress-strain curve at the point of rupture.

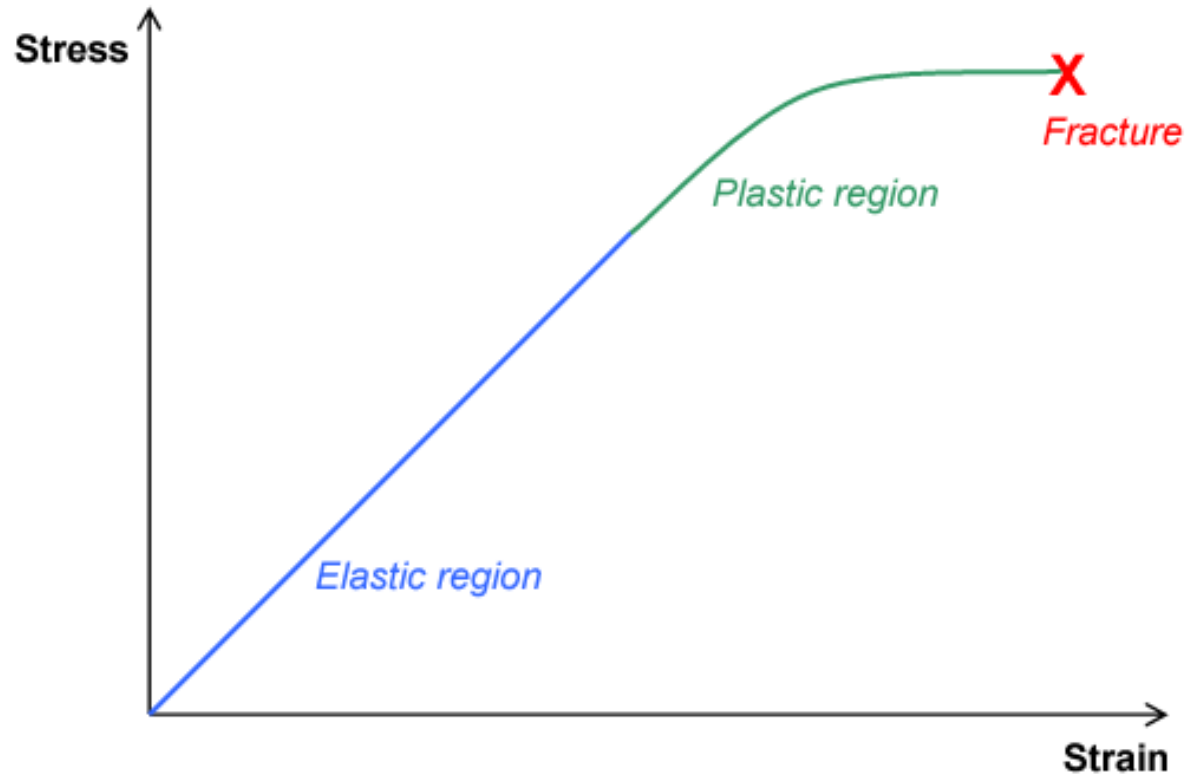
Compressive strength is the capacity of a material to withstand axially directed **pushing** forces. When the limit of compressive strength is reached, materials are crushed. Concrete can be made to have high compressive strength.

Parts and Material Selection

Tensile Strength



Another Stress-Strain Curve



Hardness

MINERAL	HARDNESS	COMMON TESTS
Talc	1	Scratched by fingernail
Gypsum	2	
Calcite	3	Scratched by copper coin
Fluorite	4	Scratched by a knife blade or window glass
Apatite	5	
Feldspar	6	Scratches a knife blade or window glass
Quartz	7	
Topaz	8	
Corundum	9	
Diamond	10	Scratches all common materials

resistance of material to the penetration of an indenter - used in analyzing service wear.

Brinell - B_{hn} (kg/mm₂)

Rockwell - R

Vickers - V_{hn}

Fatigue

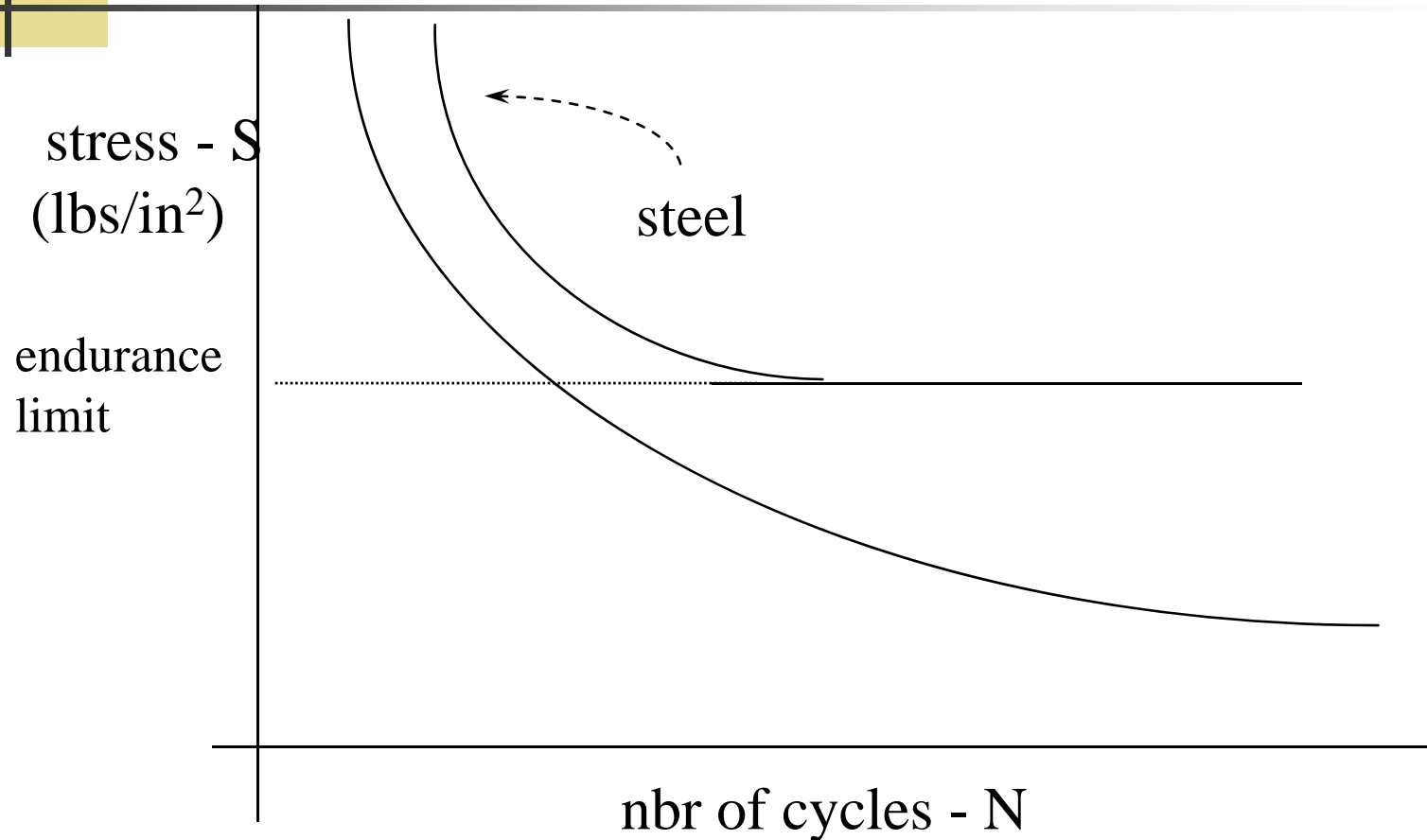


- The progressive, localized, and permanent structural damage that occurs when a material is subjected to cyclic or fluctuating strains at nominal stresses that have maximum values less than the static yield strength of the material.
 - The resulting stress may be below the ultimate tensile stress, or even the yield stress of the material, yet still cause catastrophic failure.
- In high-cycle fatigue situations, materials performance is commonly characterized by an *S-N curve*. This is a graph of the magnitude of a cyclical stress (S) against the cycles to failure (N).



Parts and Material Selection

Fatigue Life



S-N Curve: $N = c S^{-m}$ where $c, m > 0$

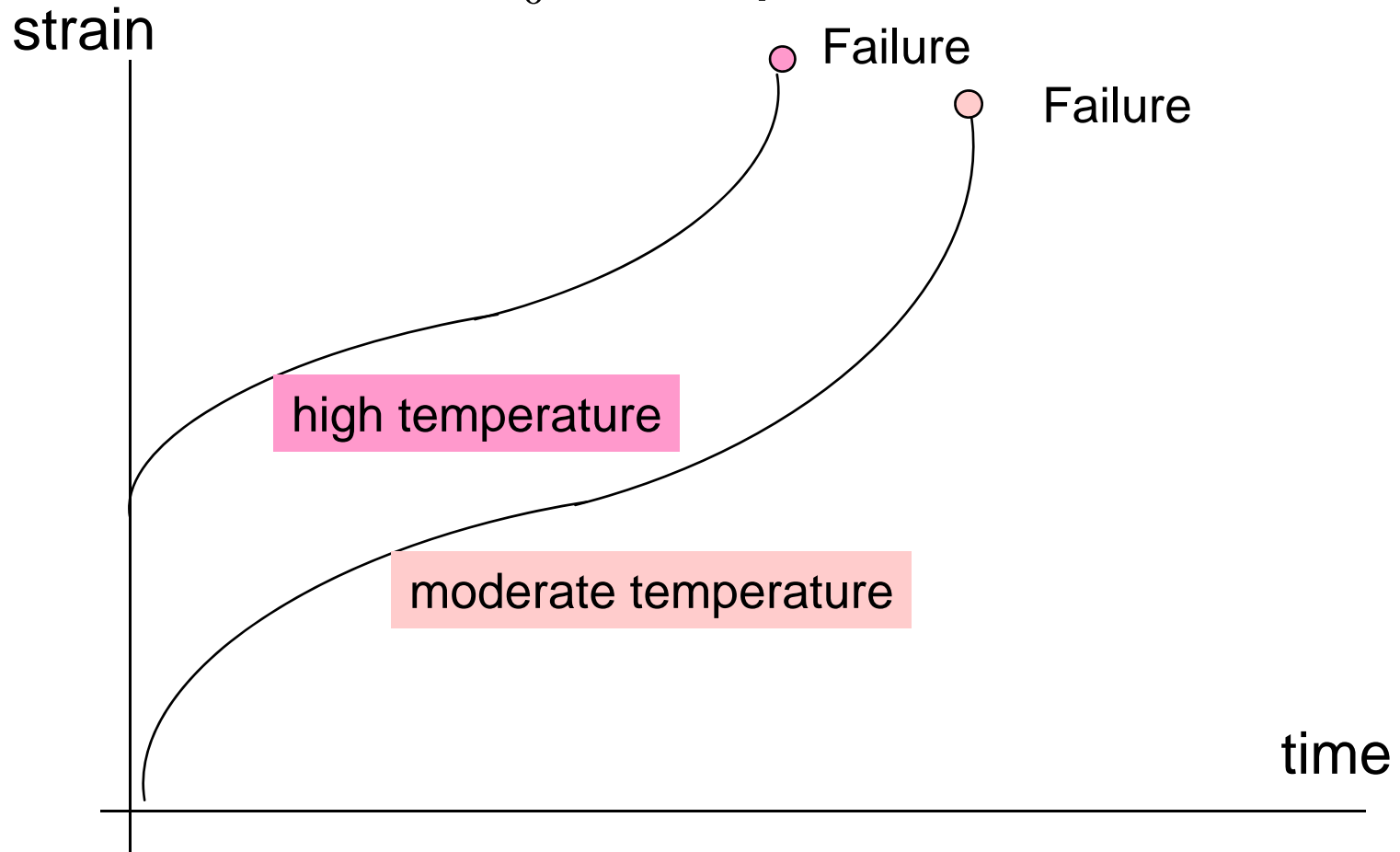


Creep

- The tendency of a material to move or to deform permanently to relieve stresses.
- Material deformation occurs as a result of long term exposure to levels of stress that are below the yield or ultimate strength of the material.
- Creep is more severe in materials that are subjected to heat for long periods and near melting point.
- The rate of damage is a function of the material properties and the exposure time, exposure temperature and the applied load (stress).

Parts and Material Selection - Creep

$$\varepsilon = \varepsilon_0 (1 + \beta t^{1/3}) e^{kt}$$





Failure modes

Failure Mode

Material Property

■ gross yielding	→	■ yield strength
■ buckling	→	■ compressive strength
■ creep	→	■ creep rate
■ brittle fracture	→	■ impact energy
■ Low cycle fatigue	→	■ ductility
■ high cycle fatigue	→	■ fatigue properties
■ corrosion	→	■ electrochemical potential
■ wear	→	■ hardness
■ thermal fatigue	→	■ coefficient of expansion



Material Failure Mechanisms

Overstress Failures

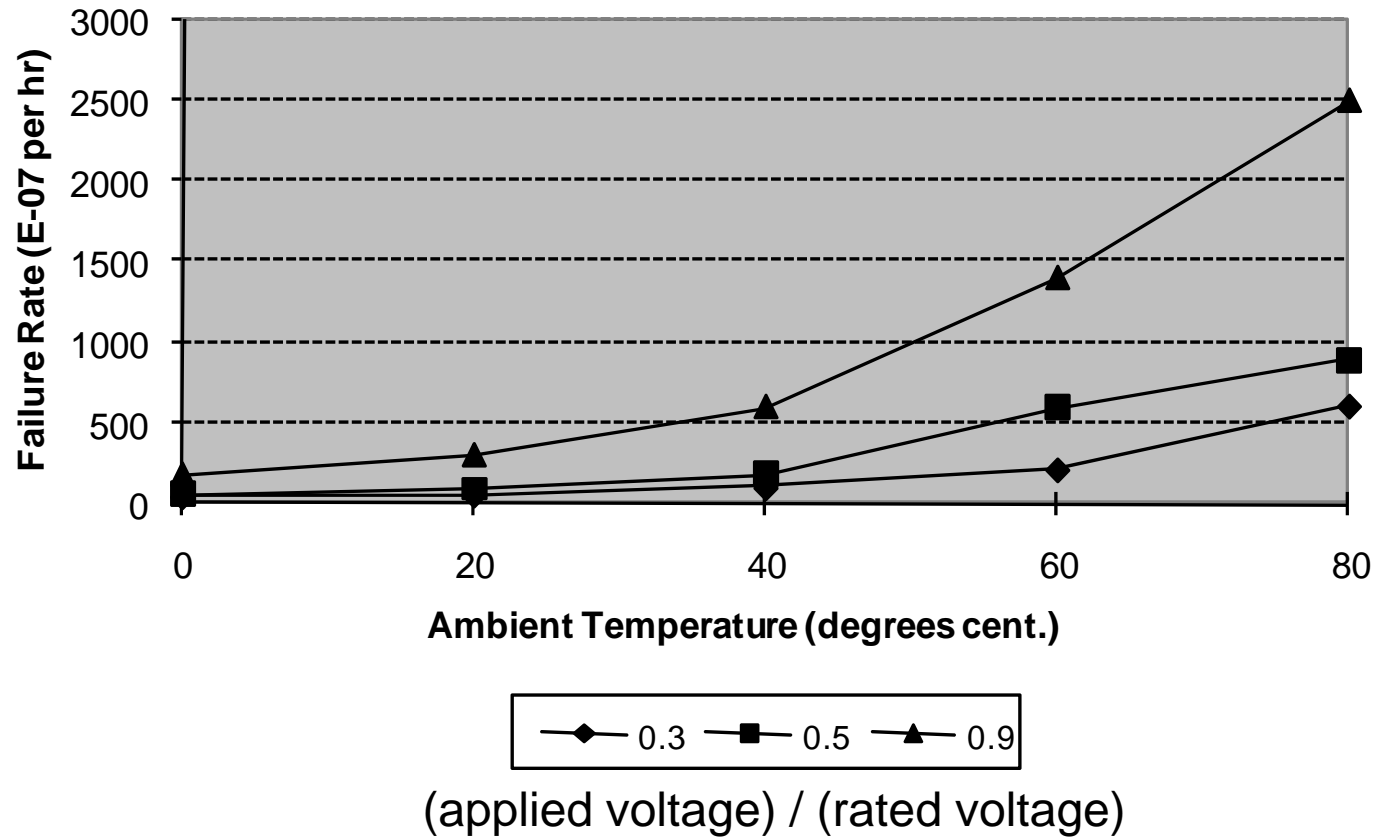
- brittle fracture
- ductile fracture
- yield
- buckling
- large elastic deformation
- thermal breakdown

Wear-out Failures

- corrosion
- dendritic growth
(electrolytic process)
- interdiffusion
- fatigue crack propagation
- diffusion (molecular migration)
- radiation
- creep
- adhesive wear

Derating

Derating Curve





More Derating

Failure Rate of a gear:
$$\lambda = \lambda_b \left(\frac{s}{s_d} \right)^{.7} \left(\frac{L}{L_d} \right)^{4.69} \left(\frac{v_s}{v} \right)^{.54} \left(\frac{c}{c_s} \right)^{.67} \left(\frac{T}{T_s} \right)^3$$

λ_b = base failure rate specified by the manufacturer

s = operating speed

s_d = design speed

L = operating load

L_d = design load

v = viscosity of lubricant used

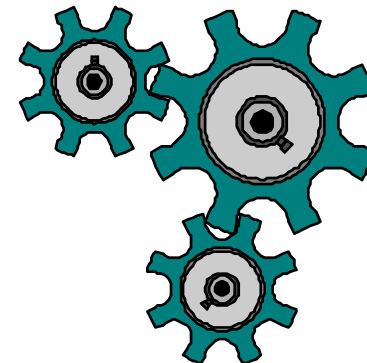
v_s = viscosity of specification lubricant

c = concentration of contaminants

c_s = standard contamination level

T = operating temperature

T_s = specification temperature





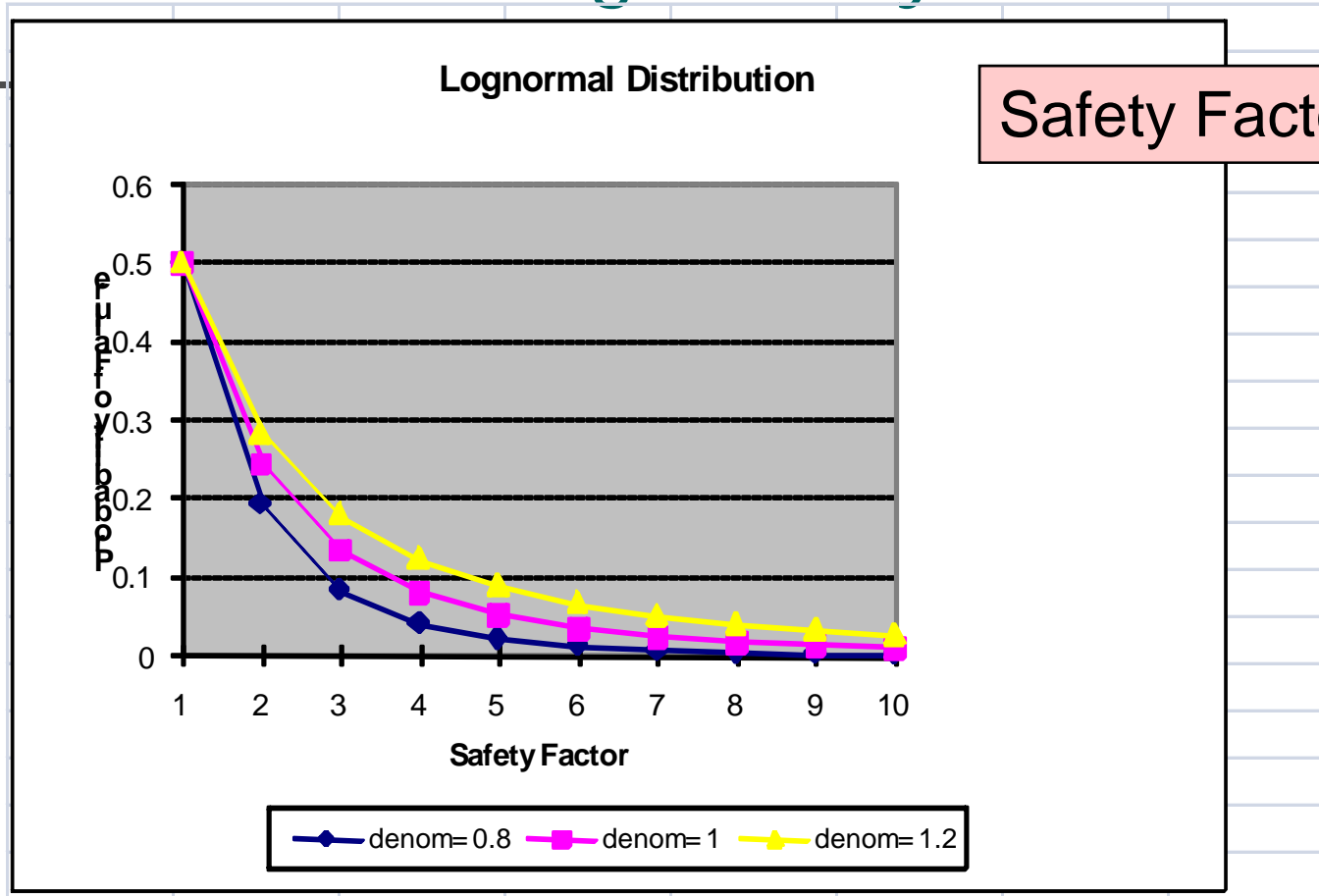
Stress-Strength Analysis

- Concerned when abnormal loads are possible
- Probabilistic compare the magnitude of the stress with the design strength.
- Use physical models
- Major categories of stress
 - electrical
 - thermal
 - mechanical
 - chemical
- Two design approaches
 - select parts with sufficient strength against max load
 - protect part against excessive stresses

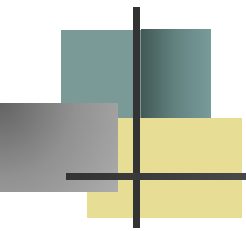


Looking for strength to
cope with the stress!

Stress - Strength Analysis



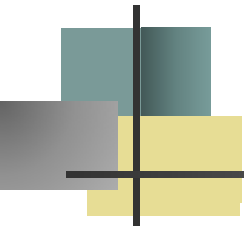
$$R = \Phi \left(\frac{\ln SF}{\sqrt{s_y^2 + s_x^2}} \right) \quad \text{where } SF = \frac{m_y}{m_x}$$



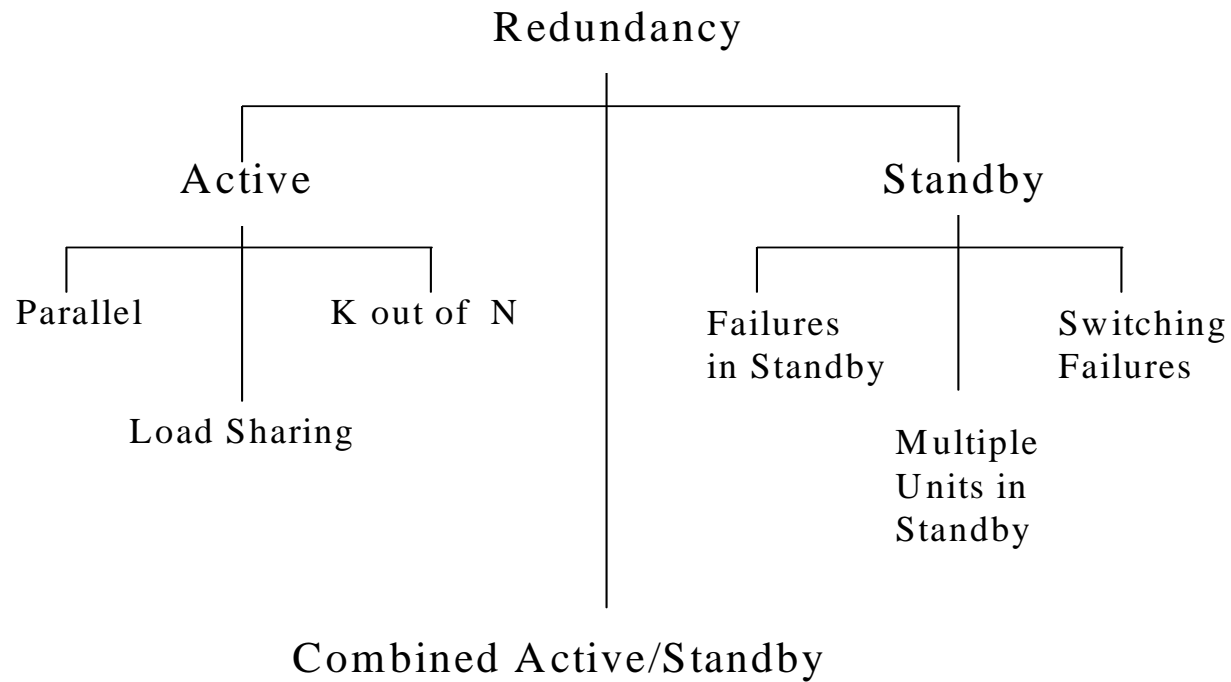
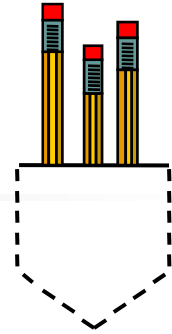
Stress Protection

Electronic Circuit Boards

Stress	Failure Mode	Design activity
high temperature	insulation deteriorates	dissipate heat, use fans, increase conductor size
thermal shock	mechanical damage	shielding
mechanical shock	component and connector damage	mechanical design - use of mountings
vibration	early wearout, connector failure	mechanical design
humidity	corrosion	sealing, use of silica gel
dust	increased contact resistance	sealing
biological effects	decayed insulation material	chemical protection



Redundancy





Redundancy

Advantages

- Quickest way to improve reliability
- May be cheapest vs. cost of redesign
- May be the only solution if specified reliability is beyond the state of the art

Disadvantages

- ⊗ Sensors and switching may increase cost and reduce reliability
- ⊗ May exceed size, weight or power constraints
- ⊗ Increases maintainability requirements

Conclusion



As an engineer, I can assure you that these design methods work.