

Rigid Body Dynamics (II)

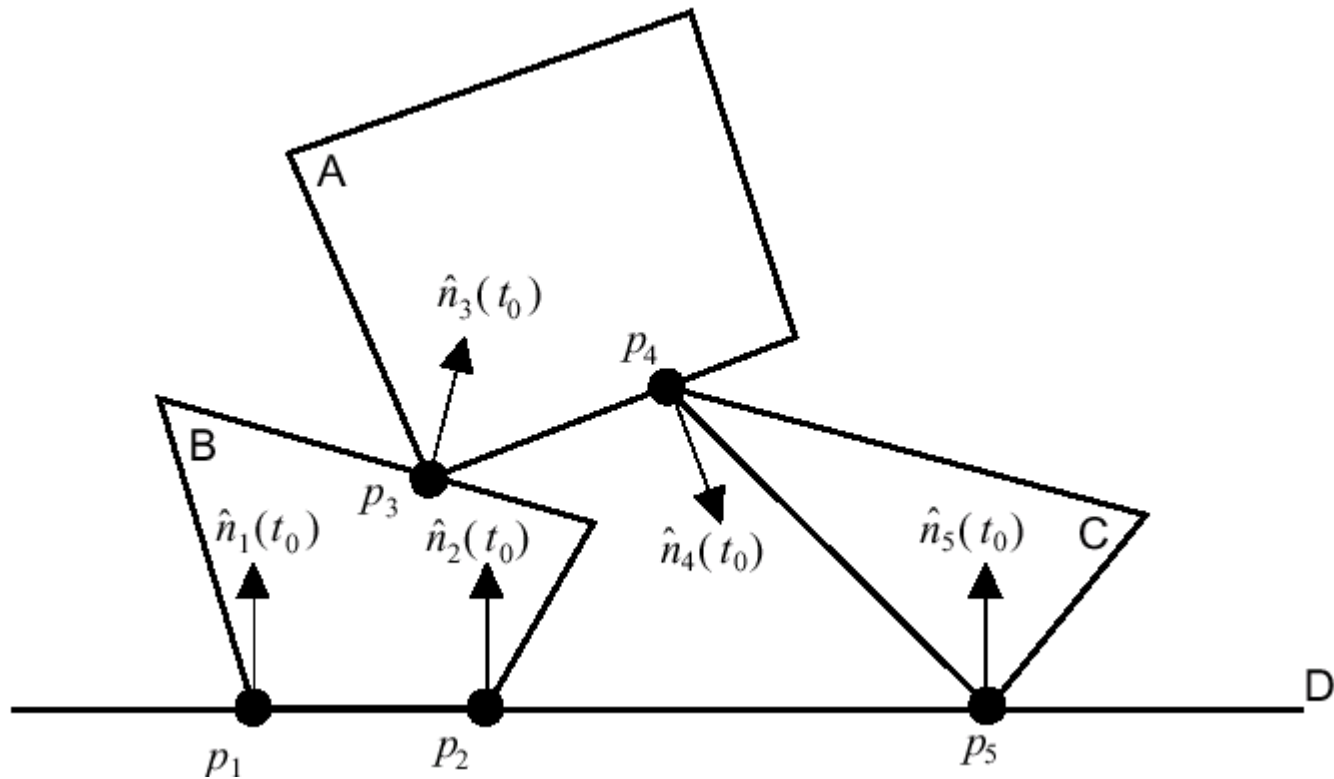
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Bodies intersect ! classify contacts

- Bodies separating
 - $v_{\text{rel}} > \varepsilon$
 - No response required
- Colliding contact (Last time)
 - $v_{\text{rel}} < -\varepsilon$
- Resting contact (Today)
 - $-\varepsilon < v_{\text{rel}} < \varepsilon$
 - Gradual contact forces avoid interpenetration
 - All resting contact forces must be computed and applied together because they can influence one another

Resting Contact Response



Handling of Resting Contact

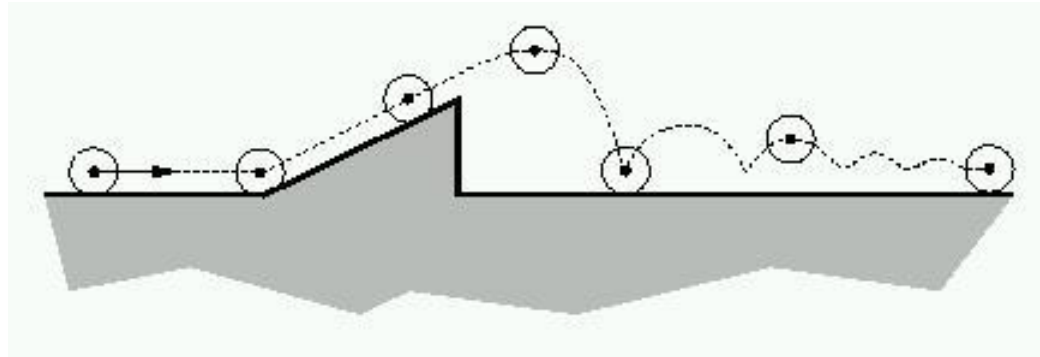
- Global vs. Local Methods
 - Constraint-based vs. Impulse-based
- Colliding Contacts
- Resting Contacts
- Force application
- Friction

Impulse vs. Constraint

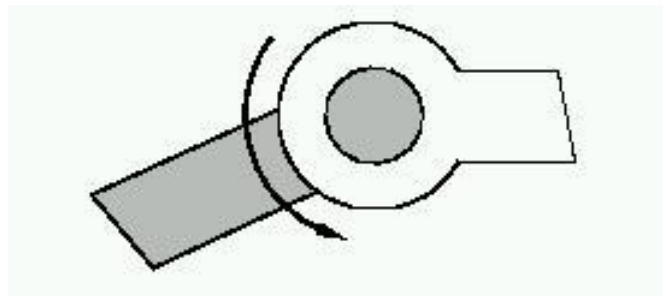
- Impulse-based dynamics (local)
 - Faster
 - Simpler
 - No explicit contact constraints
- Constraint-based dynamics (global)
 - Must declare each contact to be a resting contact or a colliding contact

Impulse vs. Constraint

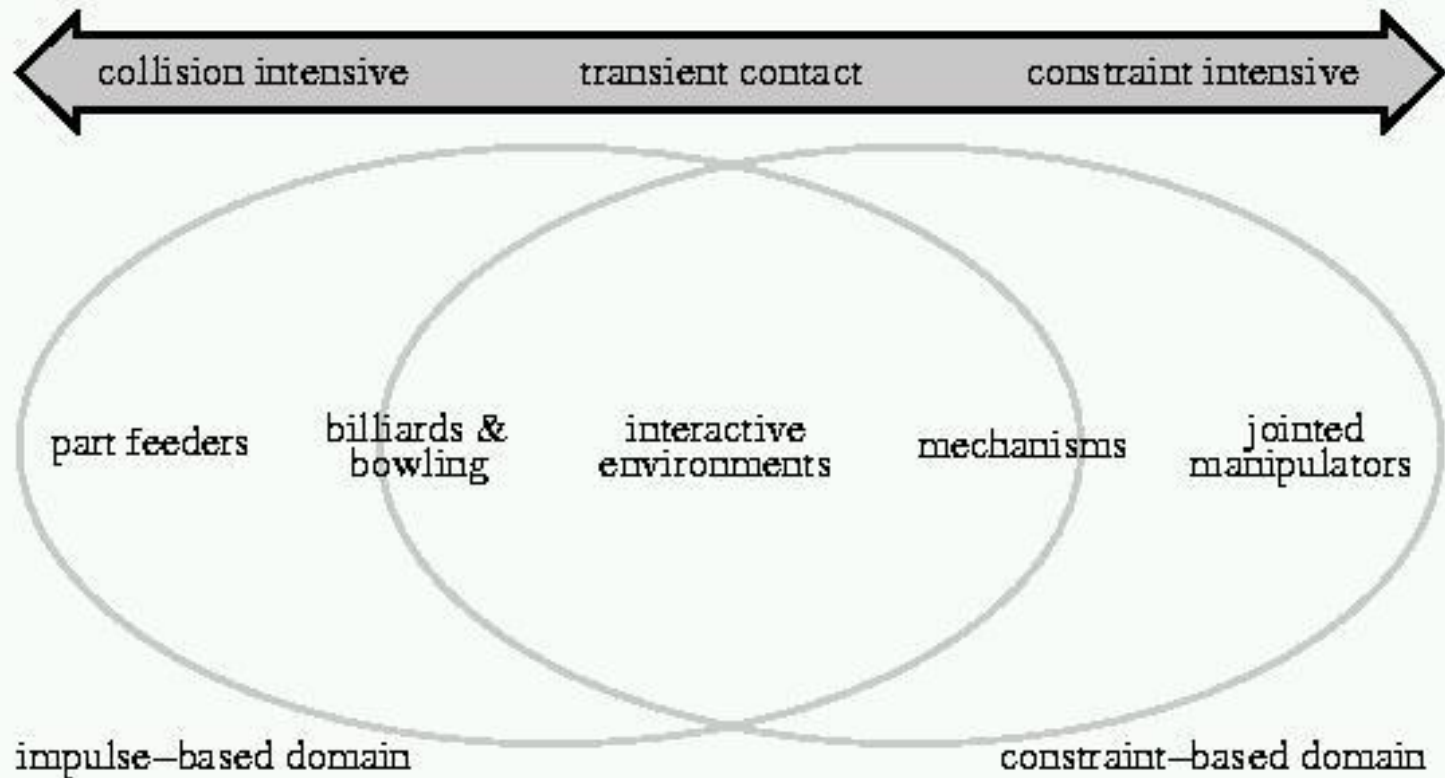
- Impulse-based dynamics (local)



- Constraint-based dynamics (global)



Impulse vs. Constraint



Resting Contact Response

- The forces at each contact must satisfy three criteria
 - Prevent inter-penetration: $\ddot{d}_i(t_0) \geq 0$
 - Repulsive -- we do not want the objects to be glued together: $f_i \geq 0$
 - Should become zero when the bodies start to separate: $f_i \ddot{d}_i(t_0) = 0$
- To implement hinges and pin joints:
$$\ddot{d}_i(t_0) = 0$$

Resting Contact Response

- We can formulate using LCP:

$$\ddot{d}_i(t_0) = a_{i1}f_1 + a_{i2}f_2 + \cdots + a_{in}f_n + b_i$$

$$\begin{pmatrix} \ddot{d}_1(t_0) \\ \vdots \\ \ddot{d}_n(t_0) \end{pmatrix} = \mathbf{A} \begin{pmatrix} f_1 \\ \vdots \\ f_n \end{pmatrix} + \begin{pmatrix} b_1 \\ \vdots \\ b_n \end{pmatrix}$$

$$\begin{cases} \ddot{d}_i(t_0) \geq 0 \\ f_i \geq 0 \end{cases} \quad f_i \ddot{d}_i(t_0) = 0$$

Linear Complimentary Problem (LCP)

- Need to solve a quadratic program to solve for the f_i 's
 - General LCP is NP-complete problem
 - A is symmetric positive semi-definite (SPD) making the solution practically possible
- There is an iterative method to solve for without using a quadratic program

[Baraff, [*Fast contact force computation for nonpenetrating rigid bodies*](#)]

Linear Complimentary Problem (LCP)

- In general, LCP can be solved with either:
 - pivoting algos (like Gauss elimination)
 - they change the matrix
 - do not provide useful intermediate result
 - may exploit sparsity well
 - iterative algos (like Conjugate Gradients)
 - only need read access to matrix
 - can stop early for approximate solution
 - faster for large matrices
 - can be warm started (ie. from previous result)

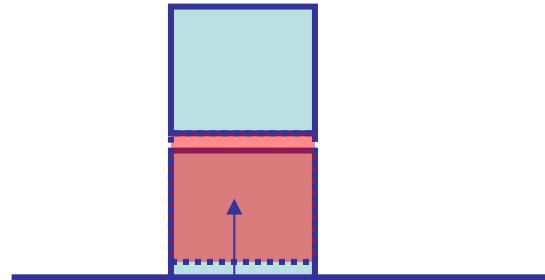
Slide courtesy of Moravanszky (ETHZ 2002)

Global vs. local?

- **Global** LCP formulation can work for either constraint-based forces or with impulses
 - Hard problem to solve
 - System very often ill-conditioned, iterative LCP solver slow to converge

Local vs. Global

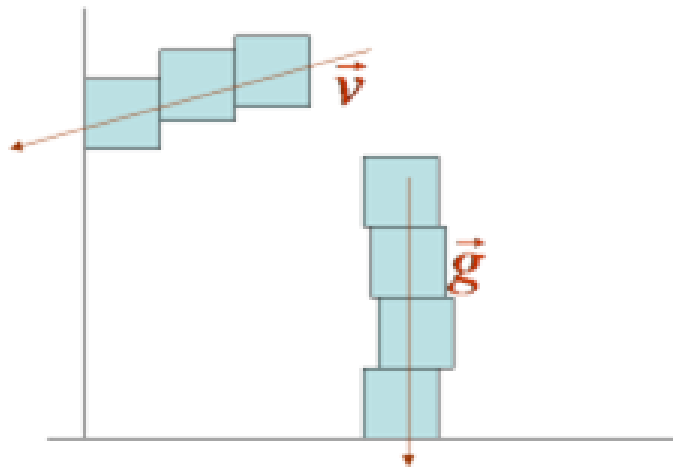
- Impulses often applied in **local** contact resolution scheme
- Applied impulses can break non-penetration constraint for other contacting points



- Often applied **iteratively**, until all resting contacts are resolved

Hard case for local approach

- **Prioritize** contact points along major axes of *acceleration (gravity)* and *velocity*
 - *Performance improvement:
25% on scene with 60 stacked objects*



Global Resting Contact Resolution

```

X  $\tilde{\leftarrow}$  InitializeState()
C  $\tilde{\leftarrow}$  Contacts( $X_{\text{new}}$ )
For t = t0 to tmax with step  $\Delta t$ 
  while (!C.isColliding())
    ClearForces(F(t),  $\tau(t)$ )
    AddExternalForces(F(t),  $\tau(t)$ )
    applyImpulses( $X_{\text{new}}$ )
     $X_{\text{new}} \tilde{\leftarrow}$  Solver::Step(X, F(t),  $\tau(t)$ , t,  $\Delta t$ )
  end if

  t  $\tilde{\leftarrow}$  findCollisionTime()
   $X_{\text{new}} \tilde{\leftarrow}$  Solver::Step(X, F(t),  $\tau(t)$ , t,  $\Delta t$ )
  restingC  $\tilde{\leftarrow}$  C.restingSet()
  C  $\leftarrow$  Contacts( $X_{\text{new}}$ )
  while (C.isColliding())
    solveCP(restingC,  $X_{\text{new}}$ )
    applyImpulses( $X_{\text{new}}$ )
  end if

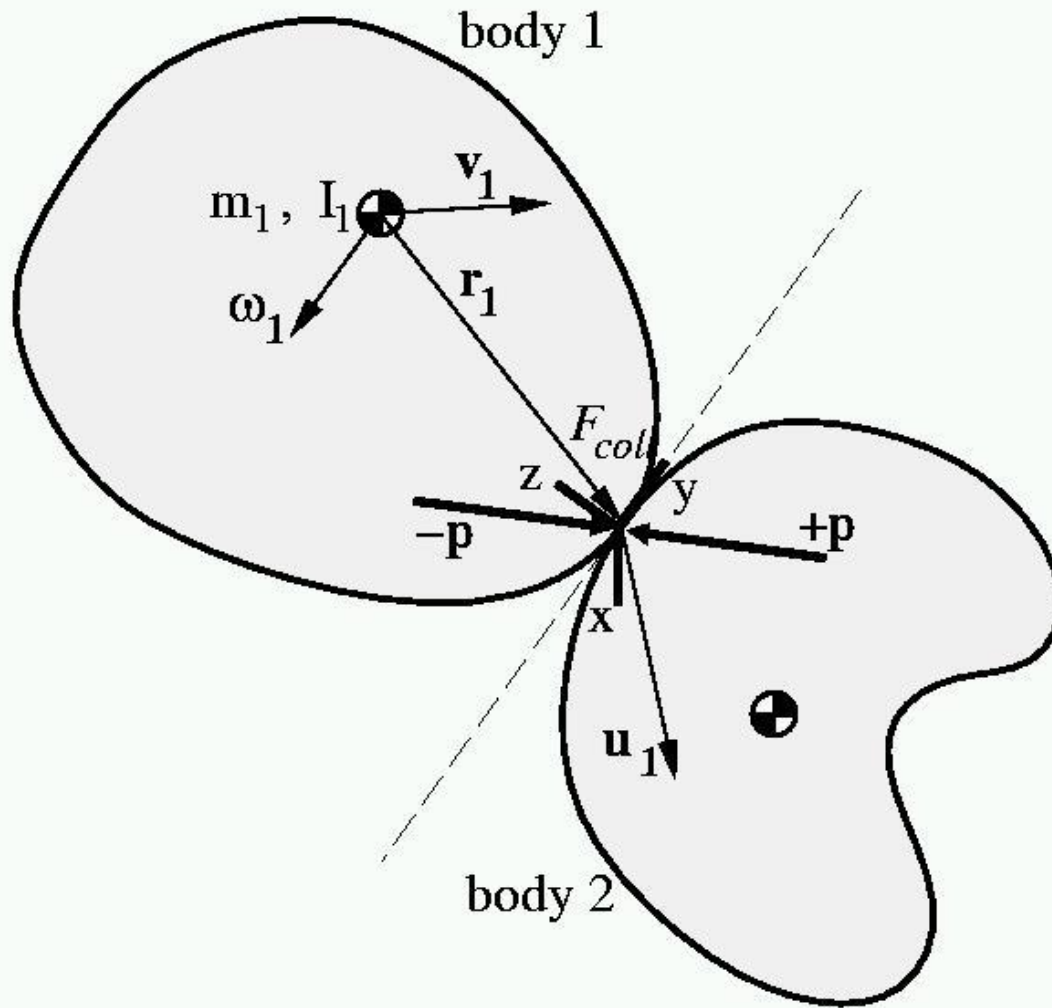
   $X \tilde{\leftarrow} X_{\text{new}}$ 
  t  $\tilde{\leftarrow}$  t +  $\Delta t$ 
End for
t  $\tilde{\leftarrow}$  t +  $\Delta t$ 
End for

```

Frictional Forces Extension

- Constraint-based dynamics
 - Reformulate constraints and solve
 - No more on this here
- Impulse-based dynamics
 - Must not add energy to the system in the presence of friction so we have to reformulate the impulse to be applied

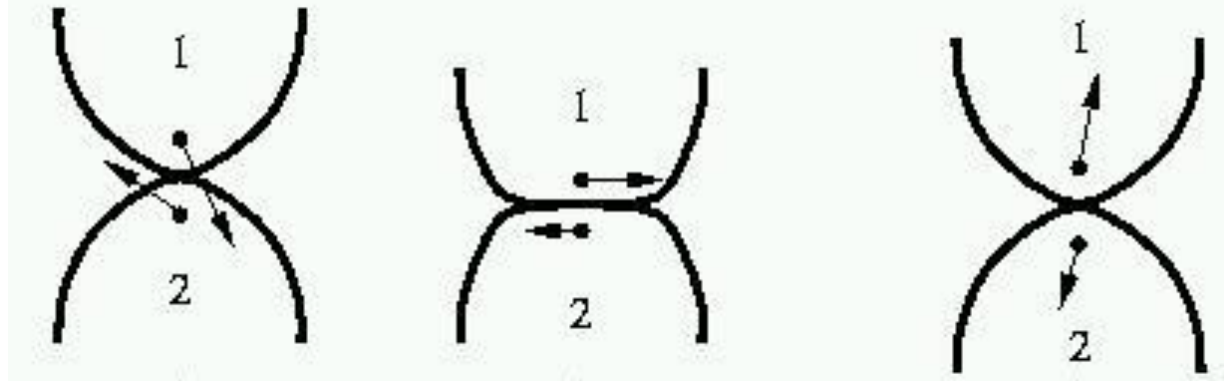
Collision Coordinate System



- \mathbf{p} is the applied impulse. We use \mathbf{j} because \mathbf{P} is for linear momentum

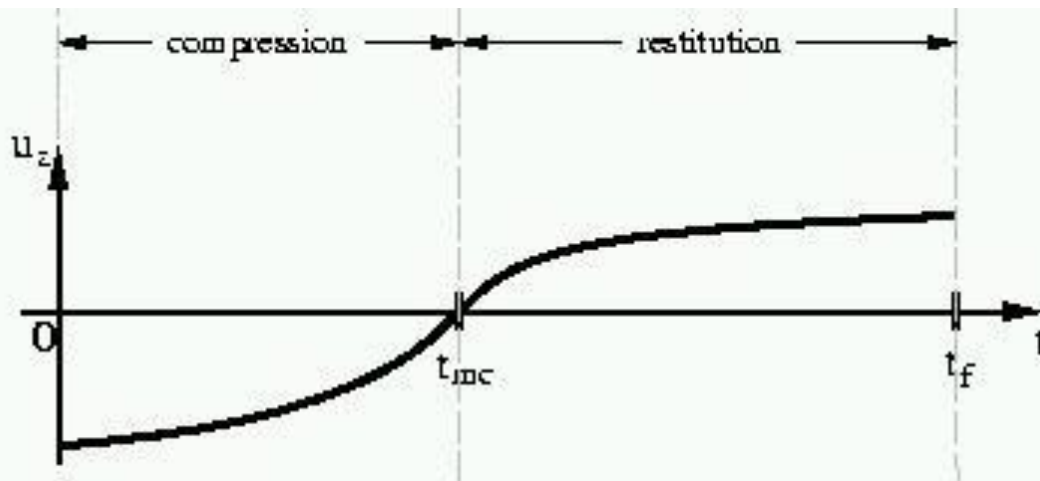
Impulse Reformulation

- When two real bodies collide there is a period of deformation during which elastic energy is stored in the bodies followed by a period of restitution during which some of this energy is returned as kinetic energy and the bodies rebound of each other.



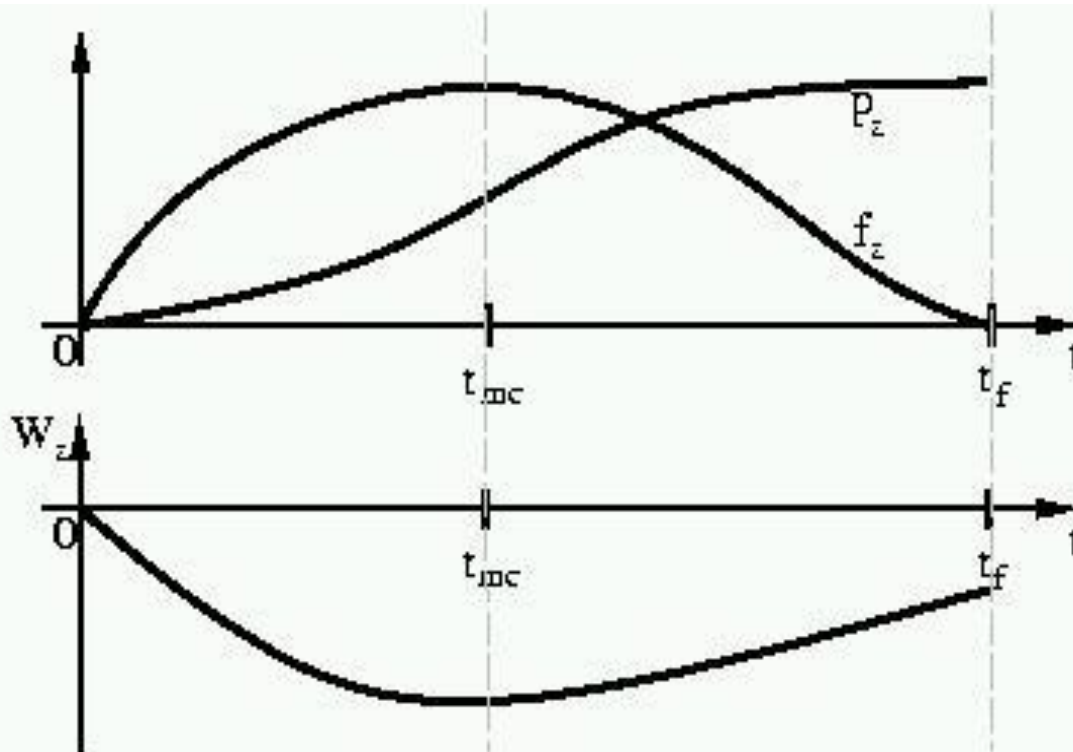
Impulse Reformulation

- The collision is instantaneous but we can assume that it occurs over a very small period of time: $0 \rightarrow t_{mc} \rightarrow t_f$.
- t_{mc} is the time of maximum compression



u_z is the relative normal velocity. We used \mathbf{v}_{rel} before. From now on we will use \mathbf{v}_z .

Impulse Reformulation



- p_z is the impulse magnitude in the normal direction. We used j before. From now on we will use j_z .
- W_z is the work done in the normal direction.

Impulse Reformulation (I)

- $v^- = v(0)$, $v^0 = v(t_{mc})$, $v^+ = v(t_f)$, $v_{rel} = v_z$
- **Newton's Empirical Impact Law:**

$$V_z^+ = -\varepsilon V_z^-$$

Coefficient of restitution ε relates before-collision to after-collision relative velocity

- **Poisson's Hypothesis:**
$$\begin{cases} j_z^+ - j_z^0 = \varepsilon j_z^0 \\ j_z^+ = (1 + \varepsilon) j_z^0 \end{cases}$$

The normal component of impulse delivered during restitution phase is ε times the normal component of impulse delivered during the compression phase

Both these hypotheses can cause increase of energy when friction is present!

Impulse Reformulation (II)

- Stronge's Hypothesis:

The positive work done during the restitution phase is $-\varepsilon^2$ times the negative work done during compression

$$\begin{cases} W_z^+ - W_z^0 = -\varepsilon^2 W_z^0 \\ W_z^+ = (1 - \varepsilon^2) W_z^0 \end{cases}$$

Energy of the bodies does not increase when friction present

Friction Formulae

- Assume the Coulomb friction law:
 - At some instant during a collision between bodies 1 and 2, let \mathbf{v} be the contact point velocity of body 1 relative to the contact point velocity of body 2. Let \mathbf{v}_t be the tangential component of \mathbf{v} and let $\hat{\mathbf{v}}_t$ be a unit vector in the direction of \mathbf{v}_t . Let \mathbf{f}_z and \mathbf{f}_t be the normal and tangential (frictional) components of force exerted by body 2 on body 1, respectively.

Coulomb Friction model

- **Sliding (dynamic) friction**

$$v_t \neq 0 \Rightarrow f_t = -\mu \|f_n\| \hat{v}_t$$

- **Dry (static) friction**

$$v_t = 0 \Rightarrow \|f_t\| \leq \mu \|f_n\|$$

(ie. the *friction cone*)

- Assume *no rolling friction*

Impulse with Friction

- Recall that the impulse looked like this for frictionless collisions:

$$j = \frac{-(1 + \epsilon)v_{rel}^-}{\frac{1}{M_a} + \frac{1}{M_b} + \hat{n}(t_0) \cdot \left(I_a^{-1}(t_0) (r_a \times \hat{n}(t_0)) \right) \times r_a + \hat{n}(t_0) \cdot \left(I_b^{-1}(t_0) (r_b \times \hat{n}(t_0)) \right) \times r_b}$$

- Remember: $p_z(t) = j(t)$ $p(t) = \int_0^t f(\tau) d\tau$
- Recall also that $\Delta v_z = j/M$ and $\Delta L = r \times j^T n$
- All are parameterized by time

Impulse with Friction

$$\Delta \mathbf{v}(t) = \left[\left(\frac{1}{m_1} \mathbf{I} + \frac{1}{m_2} \mathbf{I} \right) + (\tilde{\mathbf{r}}_1 \mathbf{I}_1^{-1} \tilde{\mathbf{r}}_1 + \tilde{\mathbf{r}}_2 \mathbf{I}_2^{-1} \tilde{\mathbf{r}}_2) \right] \mathbf{j}(t) = \mathbf{K} \mathbf{j}(t)$$

where:

$\mathbf{r} = (\mathbf{p} - \mathbf{x})$ is the vector from the center of mass to the contact point

$$\mathbf{r}^* = \begin{bmatrix} 0 & -r_z & r_y \\ r_z & 0 & -r_x \\ -r_y & r_x & 0 \end{bmatrix}$$

The K Matrix

- K is constant over the course of the collision, nonsingular, symmetric, and positive definite

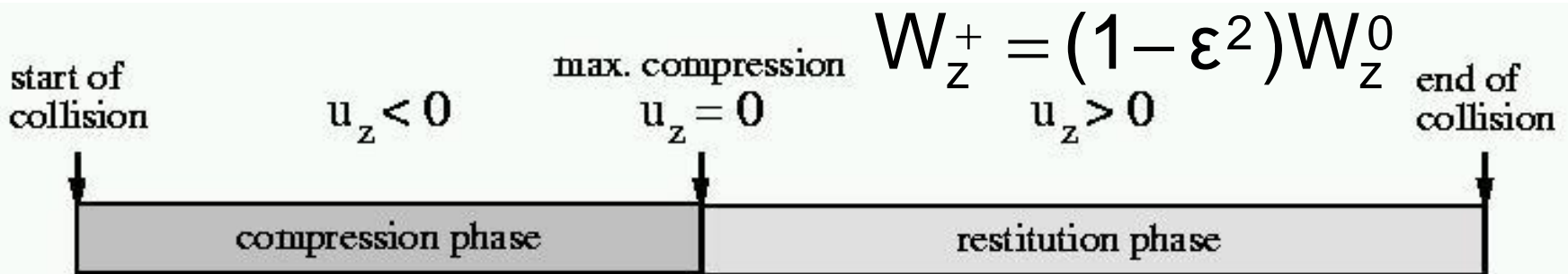
$$K = \begin{bmatrix} k_x \\ k_y \\ k_z \end{bmatrix}$$

Collision Functions

- We assume collision to occur over zero time interval ! velocities discontinuous over time
- Reparameterize $\Delta \mathbf{v}(t) = K \mathbf{j}(t)$ from \mathbf{t} to γ
- Take γ such that it is monotonically increasing during the collision: $\Delta \mathbf{v}(\gamma) = K \mathbf{j}(\gamma)$
- Let the duration of the collision $\rightarrow 0$.
- The functions \mathbf{v} , \mathbf{j} , \mathbf{W} , all evolve *continuously* over the compression and the restitution phases with respect to γ .

Sliding Formulation

- For the **compression phase**, use $\gamma = V_z$
 - V_z^- is the relative normal velocity at the start of the collision (we know this)
 - At the end of the compression phase, $V_z^0 = 0$
- For the **restitution phase**, use $\gamma = W_z$
 - W_z^0 is the amount of work that has been done in the compression phase
 - From Stronge's hypothesis, we know that



Sliding Formulation

- Compression phase equations are:

$$\frac{d}{dv_z} \begin{bmatrix} v_x \\ v_y \\ W_z \end{bmatrix} = \frac{1}{k_z \xi(\theta)} \begin{bmatrix} k_x \xi(\theta) \\ k_y \xi(\theta) \\ v_z \end{bmatrix}$$

Sliding Formulation

- Restitution phase equations are:

$$\frac{d}{dW_z} \begin{bmatrix} v_x \\ v_y \\ v_z \end{bmatrix} = \frac{1}{v_z} K \xi(\theta) = \frac{1}{v_z} \begin{bmatrix} k_x \xi(\theta) \\ k_y \xi(\theta) \\ k_z \xi(\theta) \end{bmatrix}$$

Sliding Formulation

where the sliding vector is:

$$\xi(\theta) = \begin{bmatrix} -\mu \cos \theta \\ -\mu \sin \theta \\ 1 \end{bmatrix} = \begin{bmatrix} -\mu v_x / \sqrt{v_x^2 + v_y^2} \\ -\mu v_y / \sqrt{v_x^2 + v_y^2} \\ 1 \end{bmatrix}$$

Sliding Formulation

- Notice that there is a problem at the point of maximum compression because $\mathbf{v}_z = 0$:

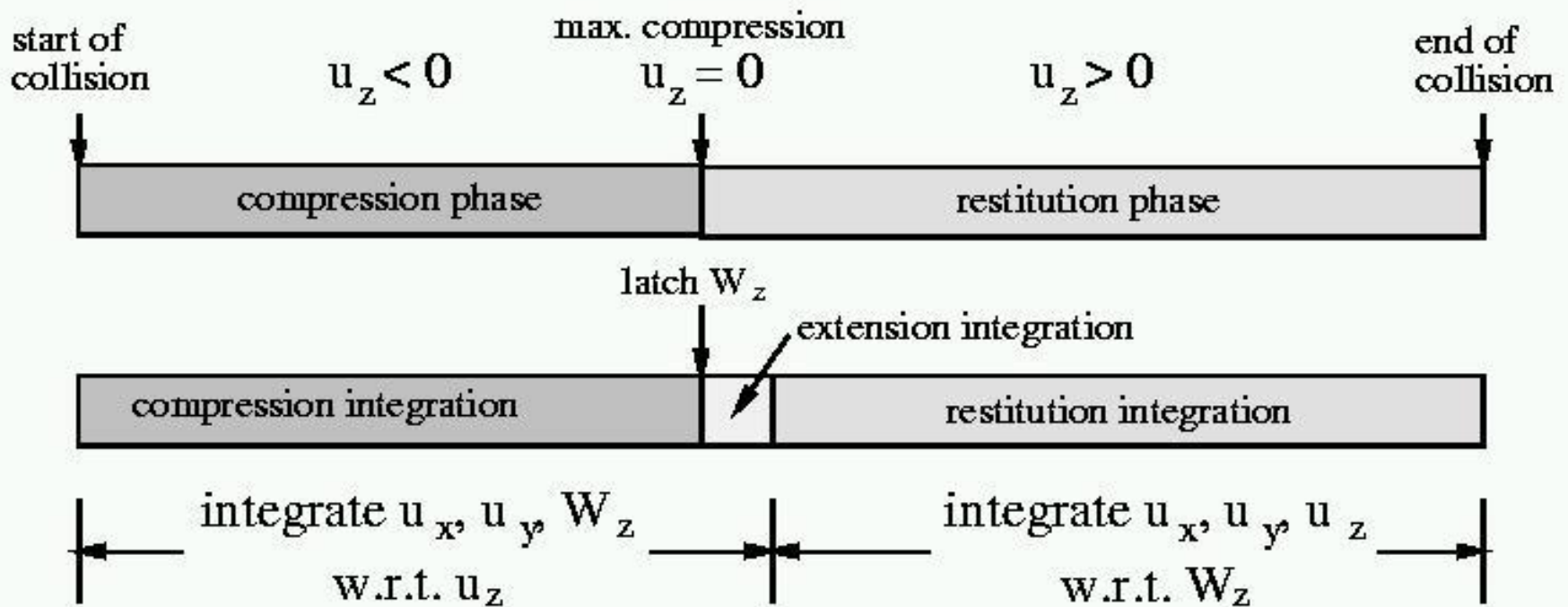
$$\frac{d}{dW_z} \begin{bmatrix} v_x \\ v_y \\ v_z \end{bmatrix} = \frac{1}{\boxed{v_z}} K \xi(\theta) = \frac{1}{\boxed{v_z}} \begin{bmatrix} k_x \xi(\theta) \\ k_y \xi(\theta) \\ k_z \xi(\theta) \end{bmatrix}$$

Sliding Formulation

- Let us integrate using \mathbf{v}_z a bit into the restitution phase (extension integration) so that we never divide by 0.
- But we don't want to integrate too far! Otherwise we exceed the amount of work that is to be done in the restitution phase.
- We are safe if we stop at:

$$\mathbf{v}_z = \sqrt{2(W_z^+ - W_z^0)(K_{33} - \mu\sqrt{K_{31}^2 + K_{32}^2})}$$

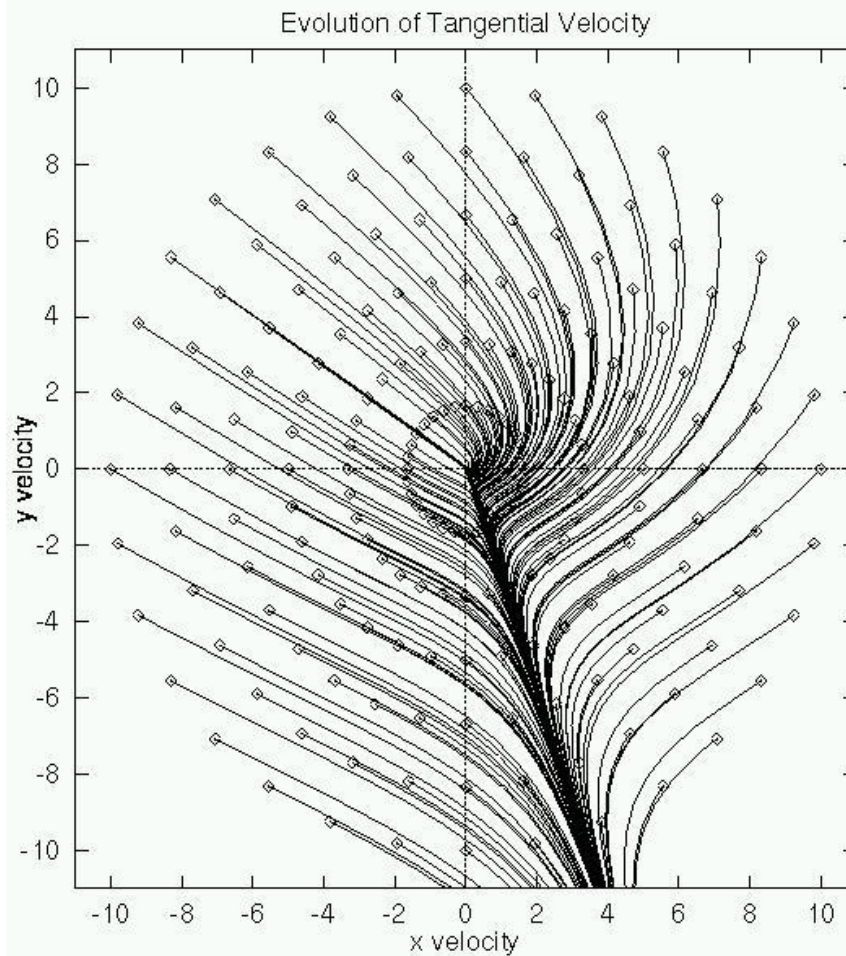
Sliding Formulation



Sliding Formulation

- There is another problem if the tangential velocity becomes 0 because the equations that we have derived were based on $v_t \neq 0 \Rightarrow f_t = -\mu \|f_n\| \hat{v}_t$ which no longer holds.
- This brings us to the sticking formulation

Sticking Formulation



Sticking Formulation

- Stable if $(K_{13}^{-1})^2 + (K_{23}^{-1})^2 \leq \mu^2 (K_{33}^{-1})^2$
 - This means that static friction takes over for the rest of the collision and \mathbf{v}_x and \mathbf{v}_y remain 0
- If instable, then in which direction do \mathbf{v}_x and \mathbf{v}_y leave the origin of the $\mathbf{v}_x, \mathbf{v}_y$ plane?
 - There is an equation in terms of the elements of K which yields 4 roots. Of the 4 only 1 corresponds to a diverging ray – a valid direction for leaving instable sticking.

Sticking Formulation

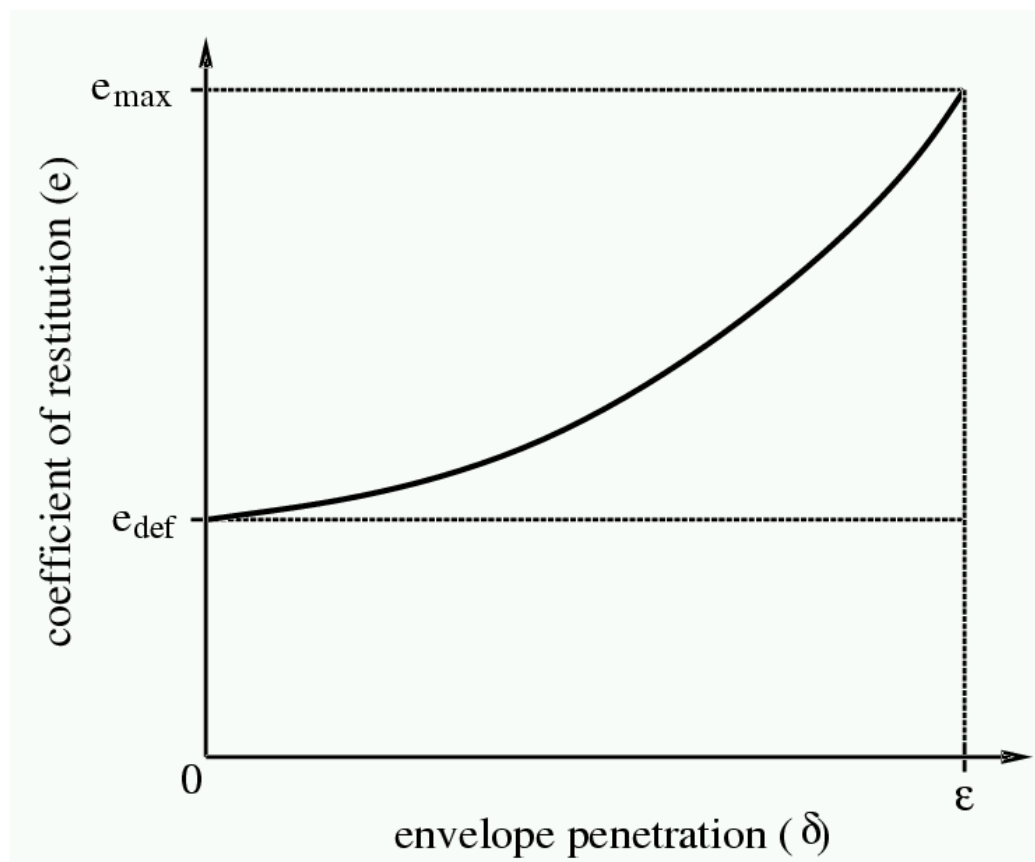
- If sticking occurs, then the remainder of the collision may be integrated analytically due to the existence of closed form solutions to the resulting simplified equations.

Resting Contacts with Impulses

- Modeled by artificial *train of collisions*
- The resulting collision impulses model a constant reaction force (do not work on stationary objects)
- Problem: book on table: through collisions, energy steadily decreases, book sinks into table
- #of collisions increases, simulator comes to grinding halt!
- Introduce *microcollisions*
 - *Microcollision impulses* are not computed in the standard way, but with artificial coefficient of restitution $\varepsilon(\delta)$
 - Applied only if normal velocity is 'small'

Artificial restitution for microcollisions

- $\epsilon = f(\text{Distance}(A,B))$



- Other problems arise:
 - Boosted elasticity from microcollisions makes box on ramp 'bounce' as if ramp were vibrating
 - Stacked books cause too many collision impulses, propagated up and down the stack
 - Weight of pile of books causes deep penetration between table and bottom book ! large reaction impulses cause instabilities
- Microcollisions are an ad-hoc solution!
- Constrained-based approaches are a better solution for these situations