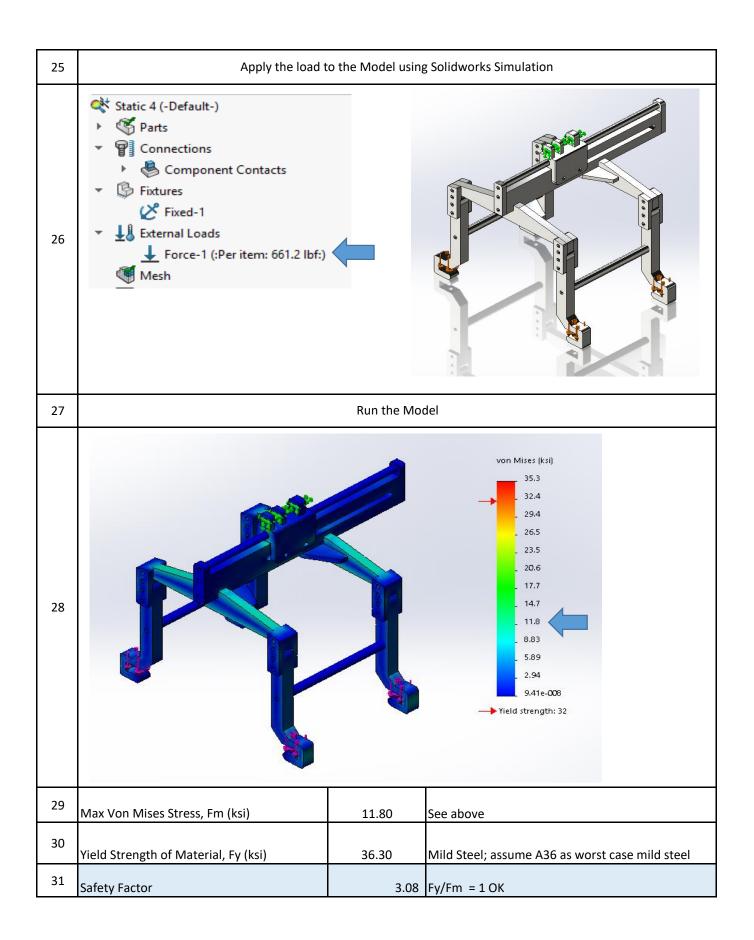


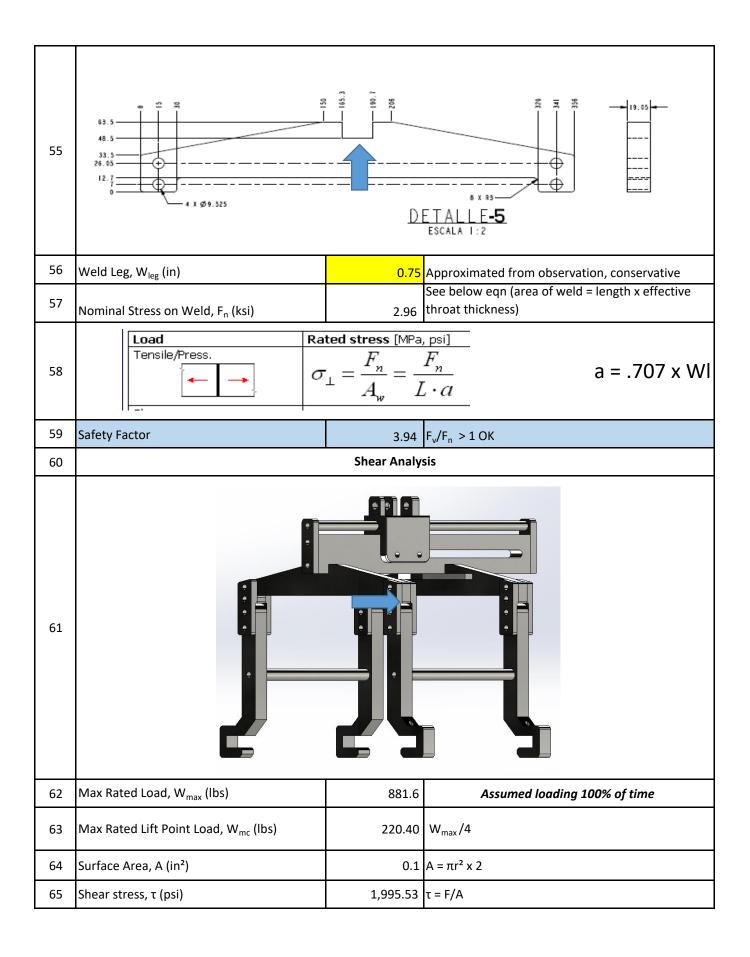
6		Drawing of Lifting Device		
7	VISTA LATER SCALE 3:16	1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	VISTA_FRONTAL SCALE 3:16	
8	Code Reference: ASME BTH-1-20	017, Below the Hoo	ok Lifting Devices; issued March 15, 2017	
9	Service Class	3	Per Drawing	
10	Table : Service Class 0 1 2 3 4	2-3-1 Service Cla Load Cyd 0-20,000 20,001-100, 100,001-500 500,001-2,00 Over 2,000,00	000 0,000 00,000	
11	Design Category	В	Lifters shall be designed to Design Category B, unless a qualified person determines that Design Category A is appropriate or that Design Category C is required for a special application.	
12	Nominal Design Factor, N _d	3	Safety Factor for Static Strength Design	
13	Nominal Design Factor, N _{dd}	3.6	The Safety Factor for Connections	
14	Design Factor = the ratio of the limit state stress (es) or strength of an element to the permissible internal stress (es) or forces created by the external force (s) that act upon the element. (section 1-2).			
15	3-1.3 Static Design Basis (17) 3-1.3.1 Nominal Design Factors. The static strength design of a below-the-hook lifting device shall be based on the allowable stresses defined in sections 3-2 and 3-3. The minimum values of the nominal design factor, N_d , in the allowable stress equations shall be as follows: $N_d = 2.00$ for Design Category A lifters $= 3.00$ for Design Category B lifters $= 6.00$ for Design Category C lifters			
16	(b) Design factors for Design Category B lifting devices shall be not less than 3.00 for limit states of yielding or buckling and 3.60 for limit states of fracture and for connection design.			
17	Job Load, J _{load} (kg)	400.0	Weight lifted by device (defined by client on drawings)	
18	Job Load, J _{load} (lbs)	881.6	J _{load} x 2.2 lbs/kg	

19	Using Solidworks, a model is created of the handling device		
20			
21	Rated Force on Upper Connection Point, F (lbs)	881.6	Job Load
22	Design factor, Df	3	For Structure
23	Applied Force, AF (lbs)	2,644.80	F x Df
24	Applied Force per Load Point Aft, (lbs)	661.20	AF/4



32	Fatigue Analysis		
33	Worst case is the top of the lifting device. (shown in solidworks simulation above). See Fatigue Stress Categories below (Ref. Table 3-4.4.1, ASME BTH-1-2017):		
34			
35	8.2 Shear on throat of continuous or intermittent longitudinal or transverse fillet welds. $ \begin{array}{c ccccccccccccccccccccccccccccccccccc$	55) Initiating at the root of the fillet weld, extending into the weld	(a) (b) (c)
36	Max Rated Load, W _{max} (lbs)	881.6	Assumed loading 100% of time
37	Max Rated Lift Point Load, W _{mc} (lbs)	220.40	W _{max} /4
38	Service Class	3	Per drawing
39	1 2 3	Load Cycles 0-20,000 20,001-100,000 100,001-500,000 500,001-2,000,000 Over 2,000,000	

45	Connection Weld Analysis -Top Lift Tab		
46			
47	(b) The design strength of fillet or partial-joint-penetration groove welds subject to shear shall be equal to the effective area of the weld multiplied by the allowable stress F_v given by eq. (3-55). Stresses in the base metal shall not exceed the limits defined in section 3-2. $F_v = \frac{0.60Exx}{1.20N_d} \tag{3-55}$		
48	Allowable Stress, F _v (ksi)	11.667 See above Eqn. 3-55	
49	Nominal Tensile Strength of weld material, E _{xx} (ksi)	70 Typical Value	
50	Table 3-3.4.3-1 Mi Material Thickness of Thick Part Joined, in. (mm) To ½ (6) Over ½ (6) to ½ (13) Over ½ (13) to ¾ (19) Over ¾ (19)	mimum Sizes of Fillet Welds ter Minimum Size of Fillet Weld, in. (mm) 1/8 (3) 3/16 (5) 1/4 (6) 5/16 (8)	
51	Tensile Load, T _n (kips)	7 Iterative until safety factor met	
52	Length of Tab, L (in)	- N/A - custom geometry	
53	Thickness of Tab, t (in)	- N/A - custom geometry	
54	Length of Fillet Weld, L (in)	25+15+15+19.05+19.05 +10+10=113.1 mm (4.45 4.45 inch)	



66	Tensile Strength of Material, f _y (psi)	58,000.0	f_{y}
67	Shear Strength of Material, v (psi)	33,466.00	0.577 x f _y
68	Shear Str. of Mat. > Shear Stress (psi)	33.5 > 2	ν>τ
69			
70	Max Rated Load, W _{max} (lbs)	881.6	Assumed loading 100% of time
71	Max Rated Lift Point Load, W _{mc} (lbs)	440.80	W _{max} /2
72	Surface Area, A (in²)	0.1	$A = \pi r^2 \times 2$
73	Shear stress, τ (psi)	3,886.73	τ = F/A
74	Tensile Strength of Material, f _y (psi)	58,000.0	f _y
75	Shear Strength of Material, v (psi)	33,466.00	0.577 x f _y
76	Shear Str. of Mat. > Shear Stress (kpsi)	33.5 > 4	ν>τ
77	The lifting device meets the allowable stress range for Service Class 3 cycle requirements.		

