

Physics for Materials and Energy Formula Booklet

1. Thermodynamics Concept

$$W = F \cdot x$$

$$P \cdot V = n \cdot R \cdot T$$

$$m = \rho \cdot V$$

$$Q = m \cdot c \cdot \Delta T$$

2. The First Law (Closed Systems)

Q is heat added to system, W is work done on system

$$E = Q + W = pe + ke + u$$

$$\dot{W} = \dot{Q} = \frac{\partial Q}{\partial t} \text{ Heat Transfer Rate}$$

$$W_{boundary} = - \int_{V_1}^{V_2} P_{ex} dV \text{ Isobaric (Constant Pressure)}$$

Isochoric (Constant Volume) = 0

$$W = m \cdot R \cdot T \cdot \ln \left(\frac{V_1}{V_2} \right) = P_1 V_1 \cdot \ln \left(\frac{V_1}{V_2} \right) \text{ Isothermal (Constant Temp)}$$

$$Q + W = E$$

3. The First Law (Open Systems)

$$h = u + P \cdot v$$

$$\dot{m} = \rho \cdot V \cdot A \text{ mass flow}$$

$$\dot{W} = \dot{m} \cdot (h_{out} - h_{in}) \text{ Steady flow devices (Turbines and Compressors)}$$

$$\dot{W} = \dot{m} \cdot [v \cdot (P_2 - P_1) + g (z_2 - z_1)] \text{ Steady flow devices (Pumps)}$$

$$V = \sqrt{2 \cdot (h_1 - h_2)} \text{ Steady flow devices (Nozzles and diffusers)}$$

When you want to calculate the change in internal energy, use c_v and when you want to calculate the change in enthalpy, use c_p

$$h(T) = h_1 + \frac{(T - T_1)}{T_2 - T_1} \cdot (h_2 - h_1) \quad \text{Interpolation for properties table}$$

$$\Delta u = c(T_2 - T_1), \Delta h = c(T_2 - T_1) + v(P_2 - P_1) \quad \text{Incompressible substance approximations}$$

h_f (enthalpy of saturated liquid) - represents the amount of energy contained in the liquid water just before it begins to vaporize.

h_g (enthalpy of saturated vapor) - indicates the amount of energy contained in the steam (vapor) when it has completely vaporized from the liquid state.

h_{fg} (enthalpy of vaporisation) - represents the amount of energy required to convert a unit mass of saturated liquid into saturated vapor at the same temperature and pressure.

4. Heat Transfer Mechanisms

$$\mathbf{q} \cdot \mathbf{n} = -h\Delta T \quad \text{Newton's Law of Cooling, } h \text{ is convection heat transfer coefficient}$$

$$\mathbf{q} = -k\Delta T \quad \text{Fourier's law of heat conduction, } k \text{ is thermal conductivity, } q \text{ is heat flux density}$$

$$R = \frac{L}{kA} \quad \text{Thermal Resistance}$$

$$G = \frac{kA}{L} \quad \text{Thermal conductance}$$

$$\|\mathbf{q}\| = \frac{\dot{Q}}{A} \quad \text{Heat flux density}$$

$$\|\nabla T\| = \frac{\Delta T}{L} \quad \text{Temperature Gradient}$$

$$\mathbf{Q} = A \cdot \epsilon \cdot \sigma \cdot T^4 \quad \text{Black-body radiation}$$

$$\dot{Q} = \frac{kA(T_1 - T_2)}{L} \quad \text{Transfer of heat through conduction}$$

5. The Second Law: Spontaneity and Reversibility. Entropy

PV Diagrams

$$\delta W = -P dV$$

$$dS = \frac{\delta Q}{T} \text{ Infinitesimally small entropy variation}$$

$$\Delta S = \int_1^2 \frac{\delta Q}{T} + S_{gen} \text{ Variation during finite process}$$

$$\Delta S = \frac{Q}{T} \text{ At constant temperature}$$

$$\Delta S = m \cdot c \cdot \ln\left(\frac{T_2}{T_1}\right) \text{ Entropy change for a reversible process (isobaric heating or cooling)}$$

$$\Delta s = c \ln\left(\frac{T_2}{T_1}\right) = c_v \ln\left(\frac{T_2}{T_1}\right) + R \ln\left(\frac{v_2}{v_1}\right) = c_p \ln\left(\frac{T_2}{T_1}\right) - R \ln\left(\frac{p_2}{p_1}\right) \text{ Entropy change for ideal gases volume form and pressure form}$$

$$\dot{S} = S_{gen} + \sum_j \left(\frac{\dot{Q}_j}{T_j} \right) + \sum_i (\dot{m}_i s_i) = S_{generated \text{ within the system}} + \text{Heat Transfer to system} \\ + \text{mass transfer to system}$$

Total entropy variation of the system

$$TV^{\gamma-1} = \text{constant}$$

$$PV^{\gamma} = \text{constant}$$

$$T_2 = T_1 \left(\frac{V_1}{V_2} \right)^{\gamma-1}, T_2 = T_1 \left(\frac{P_2}{P_1} \right)^{\frac{(\gamma-1)}{\gamma}}, \gamma = \frac{c_p}{c_v}$$

$$W = \frac{P_2 V_2 - P_1 V_1}{1 - \gamma} = \frac{mR(T_2 - T_1)}{1 - \gamma} \text{ Work in isentropic Ideal gas processes}$$

6. Phase diagrams and phase transitions

$$x = \frac{m_{vapor}}{m_{total}} = \frac{m_{vapor}}{m_{vapor} + m_{liquid}} = \left(\frac{s - s_l}{s_2 - s_l} \right) \text{ Quality (use for part vapor/liquid questions)}$$

$$y = y_f + x \cdot (y_{fg}) \text{ Interpolation for any thermodynamic property}$$

$h = h_1 + \left(\frac{s - s_1}{s_2 - s_1} \right) \cdot (h_2 - h_1)$ Example: h_1, h_2, s_1, s_2 are just chosen values from the table,
while h is the matching enthalpy for entropy s

$$h_{fg} = u_g - u_f + P(v_g - v_f) \quad \text{Latent heat of vaporization}$$

$$\frac{dP_{sat}}{dT} = \frac{h_{fg}}{T(v_g - v_f)} \quad \text{Clapeyron Equation, Describes how saturation pressure varies with temperature}$$

$$P_{sat} = C e^{\left(\frac{-\Delta h_{vap}}{RT_{sat}} \right)} \quad \text{Most common use of Clausius-Clapeyron Equation (Integrated form for P)}$$

$$\left(P + \frac{a}{v^2} \right) (v - b) = RT \quad \text{Van der Waals Equation}$$

$$Z = \frac{Pv}{RT} \quad \text{Generalized compressibility factor}$$

$$G = H - TS \quad \text{Gibbs Free energy}$$

7. Heat Engines: Rankine Cycle

$$\eta = \frac{W_{net}}{Q_{in}} = 1 - \frac{Q_C}{Q_H} \quad \text{Thermal Efficiency of Thermal Engines}$$

1. Carnot Cycle

$$\eta_{carnot} = 1 - \frac{T_C}{T_H}$$

$$\frac{Q_H}{T_H} = \frac{Q_C}{T_C}$$

2. Rankine Cycle

$$\eta_{rankine} = \frac{\dot{W}_{turb} - \dot{W}_{pump}}{\dot{Q}_{in}}$$

$$\dot{W}_{turb} = \dot{m}(h_3 - h_4)$$

$$\dot{W}_{pump} = \dot{m}v(P_2 - P_1)$$

$$\dot{Q}_{in} = \dot{m}(h_3 - h_2)$$

8. Refrigeration cycles

$$COP_{carnot, ref} = \frac{T_C}{T_H - T_C}$$

$$COP_{carnot, hp} = \frac{T_H}{T_C - T_H}$$

$$Q_H = Q_C + W \Rightarrow Q_C = Q_H - W_{net}$$

$$COP_{ref} = \frac{Q_C}{W} = \frac{h_1 - h_4}{h_2 - h_1}$$

$$COP_{hp} = \frac{Q_H}{W} = \frac{Q_C + W}{W}$$

9. Gas Power Cycles

Otto Cycle

1>2 Isentropic Compression, 2>3 Constant Volume heat addition, 3>4 Isentropic Expansion, 4>1 Constant Volume heat rejection

$$r = \frac{V_1}{V_2} = \frac{V_{max}}{V_{min}} \quad \text{Compression Ratio}$$

$$\eta_{th, otto} = 1 - \frac{1}{r^{\gamma-1}} \quad \text{Thermal Efficiency}$$

Diesel Cycle

1>2 Isentropic Compression, 2>3 Constant pressure heat addition, 3>4 Isentropic Expansion, 4>1 Constant Volume heat rejection

$$r_c = \frac{V_3}{V_4} \quad \text{cut-off ratio}$$

$$\eta_{th, diesel} = 1 - \frac{1}{r^{\gamma-1}} \cdot \frac{r_c^{\gamma} - 1}{\gamma(r_c - 1)} \quad \text{Thermal Efficiency}$$

$$Q_H = h_3 - h_2 \quad \text{Heat added}$$

Brayton Cycle

1>2 Isentropic Compression, 2>3 Constant pressure heat addition, 3>4 Isentropic Expansion, 4>1 Constant Pressure heat rejection

$$bwr = \frac{w_{comp}}{w_{turb}} = \frac{h_2 - h_1}{h_3 - h_4} \quad \text{Back Work Ratio}$$

$$\eta_{th, brayton} = 1 - \left(\frac{T_1}{T_2} \right)^{\frac{\gamma-1}{\gamma}} \quad \text{or } \eta = \frac{w_{net}}{q_{in}} \quad \text{Thermal Efficiency}$$

10. Gas Power Cycles