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%Section - 01
%Aero 431 HW4: 6/1/25
```

Workspace Prep

Question 1

```
% Constants
E = 70e9;
                     % Young modulus [Pa]
G = 26e9;
                     % Shear modulus [Pa]
sig y = 280e6;
                    % Yield strength [Pa]
nu = (E / (2 * G)) - 1; % Poisson ratio
t = 2e-3;
                    % Thickness [m]
T = 5e-3;
                    % Flange thickness [m]
h = 0.1;
                     % Web height [m]
                     % Flange width [m]
w = 0.05;
% Initiailize L and b
L = linspace(0.1, 0.5, 6);
b = linspace(0.1, 0.3, 6);
% Simply supported plate buckling coefficient
k plate = @(AR, m) (pi^2) * (m^2 + (AR * m)^2);
% Simply supported plate buckling stress
buckling stress = @(E, k, t, b) (k * pi^2 * E / (12 * (1 - nu^2))) * (t / pi^2)
b)^2;
% Prepare result array
results = [];
```

```
% For loop to iterate through all L and b possibilities
for i = 1:length(L )
for j = 1:length(b)
    L = L (i);
   b = b (j);
   mode = 1;
    % Calculate aspect ratios for each structural element
   AR skin = L / b;
    AR web = h / b;
    AR flange = L / (w / 2);
   AR Iplate = h / b;
    % Calculate respective buckling coefficients from aspect ratio
    k skin = k plate(AR skin, mode);
    k web = k plate(AR web, mode);
    k flange = k plate(AR flange, mode);
    k Iplate = k plate(AR Iplate, mode);
    % Calculate buckling stress for each element
    sigma skin = min(buckling stress(E, k skin, t, b), sig y);
    sigma web = min(buckling stress(E, k web, t, b), sig y);
    sigma flange = min(buckling stress(E, k flange, T, w/2), sig y);
    sigma Iplate = min(buckling stress(E, k Iplate, T, b), sig y);
        I_beam = 2 * (w * T^3) / 12 + 2 * w * T * (h/2)^2;
        A beam = (2 * w * T) + (h * t);
    sigma Euler = min((pi^2 * E * I beam) / (A beam * L^2), sig y);
    % Minimum stresses as critical stress value
    sigmas plate = [sigma skin, sigma web, sigma flange, sigma Iplate];
    sigmas Ibeam = [sigma skin, sigma web, sigma flange, sigma Euler];
    [sigma crit plate, i plate] = min(sigmas plate);
    [sigma crit Ibeam, i Ibeam] = min(sigmas Ibeam);
    components = {'Skin panel', 'Spar web', 'Spar flange', 'I-beam plate'};
    if i Ibeam == 4
        components{4} = 'I-beam';
    end
    % Store results
    results = [results;
        L, b, ...
        sigma skin, sigma web, sigma flange, sigma Iplate, sigma Euler, ...
        sigma crit plate, sigma crit Ibeam, ...
        i plate, i Ibeam];
end
end
```

```
% Display as table
headers = {'L [m]', 'b [m]', 'sig skin [Pa]', 'sig web [Pa]', 'sig flange
            'sig Iplate [Pa]', 'sig Euler [Pa]', 'sig crit plate [Pa]', ...
            'sig crit Ibeam [Pa]', 'Mode Plate Idx', 'Mode Ibeam Idx'};
% Organize results
solution = array2table(results, 'VariableNames', headers);
mode names = {'Skin Panel', 'Spar Web', 'Spar Flange', 'I-beam Plate/Beam'};
solution.Mode Plate = mode names(solution.Mode Plate Idx)';
solution.Mode Ibeam = mode names(solution.Mode Ibeam Idx)';
% Display results
disp(solution)
% The mode numbers seem to vary slightly, & the buckling stress seems to
% decreases with rib spacing. It was also oberved that the panel was the
% first to buckle. Increasing rib / spar count will improve strength.
[Pa] sig crit Ibeam [Pa] Mode Plate Idx Mode Ibeam Idx

    0.1
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    280000000

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280000000 1
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      0.26
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           {'Spar Web' } {'Spar Web' }
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      198969072.113945

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    0.42
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    0.42
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    247805870.118362
    64376613.9453395

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      274065489.914672
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      198969072.113945

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      104299407.776978
      104299407.776978

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280000000
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5

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64376613.9453395 64376613.9453395 2
2 {'Spar Web' } {'Spar Web' }
0.5 0.26 179470514.632783 43850478.3863475
280000000 274065489.914672 280000000
43850478.3863475 43850478.3863475 2
2 {'Spar Web' } {'Spar Web' }
0.5 0.3 108392667.553886 31880196.3393782
280000000 199251227.121114 280000000
31880196.3393782 31880196.3393782 2
2 {'Spar Web' } {'Spar Web' }
```

Problem 2

```
% Part 1
syms a % crack length [m]
w = 0.5; % specimen width [m]
sigma = 50e6; % applied stress [Pa]
% Define alpha = a/W
alpha = a / w;
% Define the beta function (geometry factor)
beta = (1.122 - 1.122*alpha - 0.820*alpha^2 + 3.768*alpha^3 -
3.040*alpha^4) / sqrt(1 - 2*alpha);
% Stress intensity factor K I
K1 = beta * sigma * sqrt(pi * a); % [Pa·sqrt(m)]
K1c = 24e6; % MPa*sqrt(m)
eqn = K1 == K1c;
a = min(real(double(solve(eqn, a))));
disp("Critical Crack Length: " + a +" m")
% Part 2
m = 3.59;
C = 3.15e-11;
sig max = 50e6;
                     % Maximum stress [Pa]
a0 = a/2;
                     % Initial crack length [m]
ac = a0*2;
dsig = 50; % MPa (keep these units for emirical relation)
dN = 100;
Ncycles = 0; % Initialize
Ncycles2 = 0;
a = a0;
while a < ac
    alpha1 = a / w;
```

```
beta1 = (1.122 - 1.122*alpha1- 0.820*alpha1^2 + 3.768*alpha1^3 -
3.040*alpha1^4) / sqrt(1 - 2*alpha1);
    dK = beta1*dsig*sqrt(pi*a);
    a = a + C*dN*dK^m;
    Ncycles = Ncycles + dN;
end
disp("Number of Cycles(dN = 100): " + Ncycles)
dN = 10;
a = a0;
while a < ac
    alpha1 = a / w;
   beta1 = (1.122 - 1.122*alpha1 - 0.820*alpha1^2 + 3.768*alpha1^3 -
3.040*alpha1^4) / sqrt(1 - 2*alpha1);
    dK = beta1*dsig*sqrt(pi*a);
    a = a + C*dN*dK^m;
    Ncycles2 = Ncycles2 + dN;
end
disp("Number of Cycles(dN = 10): " + Ncycles2)
% Comment:
% The difference between the dN = 100 and dN = 10 cycles is not very large.
% (only about 80 cycles, which is negligible)
Critical Crack Length: 0.05792 m
Number of Cycles (dN = 100): 19100
Number of Cycles (dN = 10): 19020
```

Question 3

```
% i) With a paper with a central crack, the size of the crack seems to
% dictate the critical stress value that the paper can hold before the
% crack propagates. Additionally, the sheet of paper fails by the crack
% growing from the center of the paper towards the edges. We found that
% the tear was very uniform and continued in the direction of the crack
% nearly perfectly.
% ii) The folded paper with flanges/stringer design and a central crack
% appears to be more resistant to crack propagation and failure from
% tensile stress. The crack of the folded paper took more tensile force
% to grow than the flat sheet of paper. This means that the folded paper
% has a greater critical tensile stress than the flat paper. The crack
% still propagates from the inside to the outside similar to the flat
% paper, and maintains the same direction as the original central crack.
% The tear appears slightly more straight and uniform than the first trial
% with the flat sheet of paper, this is most likely due to the increase in
% rigidity from the bends in the paper or "stringers."
% iii) We conducted the experiment with each configuration one time:
```

```
Configuration 1: narrow glue surface, ~1 cm wide
      Configuration 2: wide glue surface, ~3 cm wide
% We found that the wider glue surface caused the tensile load to be
% better transferred between each sheet of paper and increase the overall
% resistance to the crack propagation.
      Configuration 3: weak glue
      Configuration 4: strong glue
% Having a higher quality, or stronger, glue had more resistance to crack
% propagation. This is a similar result as having a wider glue surface.
% We suspect that the strength of the bond between sheets of paper is
% directly related to the resistance against tearing. The bond strength
% is increased whether the strength of the glue is increased or the
% surface area of the bond is increased.
% We found that for each configuration, after the center sheet of paper
% completely ripped, the glue would then fail and the outside sheets of
% paper would not tear.
% iv) We conducted the experiment with each configuration one time:
      Configuration 1: pins close to the edge, ~1 cm
      Configuration 2: pins far from the edge, ~3 cm
% We noticed that having pins closer to the edge of the paper causes the
% edge of the paper to rip near the fasteners at smaller tensile loads.
       Configuration 3: less frequent pin placement
      Configuration 4: more frequent pin placement
% Having more pins along the fastening surface increased the resistance to
% crack propagation and increased the critical stress of the paper around
% the fasteners. More pins means the load can be distributed to more
% fasteners which lowers the individual stress on each one, meaning the
% system as a whole can take a higher load.
```

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