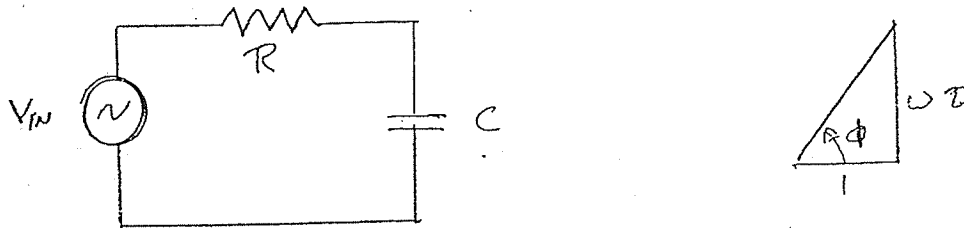


## E-84 Problem Set #6 Key

### Problems from Chapter 5

1. a. Show that for a sinusoidal input voltage  $v_{IN} = A \sin \omega t$  the steady state solution for the voltage across the capacitor in the R-C circuit shown below is

$$v_{ss}(t) = \frac{A \sin(\omega t - \phi)}{\sqrt{1 + \omega^2 \tau^2}} = \frac{A \sin \omega(t - t_{lag})}{\sqrt{1 + \omega^2 \tau^2}} \text{ where } \phi = \tan^{-1} \omega \tau \text{ and } \tau = RC$$



Thus the voltage on the capacitor oscillates at the same frequency as the source, but with a smaller amplitude and a time lag,  $t_{lag} = \phi/\omega$ .

An easy way to get the particular (steady state) solution is to use circuit theory with phasors, so that

$$\underline{V}_C = \underline{V} Z_C / (Z_C + R) = \underline{V} / (1 + j\omega RC) = V / (1 + \omega^2 \tau^2)^{1/2} \angle -\phi$$

where  $\tau = RC$  and  $\phi = \tan^{-1} \omega RC$

Hence for  $v(t) = A \sin \omega t$ ,

$$v_{ss}(t) = \frac{A \sin(\omega t - \phi)}{\sqrt{1 + \omega^2 \tau^2}} = \frac{A \sin \omega(t - t_{lag})}{\sqrt{1 + \omega^2 \tau^2}} \text{ where } \phi = \tan^{-1} \omega \tau \text{ and } \tau = RC, \text{ and}$$

$$t_{lag} = \phi/\omega = (1/\omega) \cdot \tan^{-1} \omega RC$$

The same result is obtained by solving for the particular solution to the governing equation for this system:

$$iR + v_C = v_{IN} \text{ or } RC(dv_C/dt) + v_C = A \sin \omega t$$

b. Show that for low frequency oscillations, when  $\omega \tau \ll 1$ , the time lag  $t_{lag} \approx \tau$ , while for high frequency oscillations when  $\omega \tau \gg 1$ , the time lag  $t_{lag} \approx T/4$ , where  $T = 1/f = 2\pi/\omega$  is the period of the oscillations.

We are given

$$v_{ss}(t) = \frac{A \sin(\omega t - \phi)}{\sqrt{1 + \omega^2 \tau^2}} = \frac{A \sin \omega(t - t_{lag})}{\sqrt{1 + \omega^2 \tau^2}} \text{ where } t_{lag} = \phi/\omega \text{ and } \phi = \tan^{-1}(\omega \tau)$$

Note that

$\tan^{-1}x \approx x$  for  $x \ll 1$ , and  $\tan^{-1}x \approx \pi/2$  for  $x \gg 1$ . Hence  
for  $\omega\tau \ll 1$ ,  $\phi \approx \omega\tau$  so that the time lag  $t_{\text{lag}} \approx \tau$ .

For  $\omega\tau \gg 1$ ,  $\phi \approx \pi/2$  so that the time lag  $t_{\text{lag}} \approx \pi/2\omega = T/4$ ,  
where  $T = 1/f = 2\pi/\omega$  is the period of the oscillations.

c. Show that the complete solution, including the homogeneous (or transient) term is

$$v_C(t) = v_{ss}(t) + Ce^{-t/\tau}, \text{ where, for } v_C(0) = 0, C = (A \sin \phi) / (1 + \omega^2 \tau^2)^{1/2}$$

As shown in the notes, the homogenous solution for the R-C circuit is  $v_{CH} = Ce^{-t/\tau}$ . To determine C we use the initial condition for the entire solution,  $v_C(0) = 0$ , to find:

$$0 = \frac{-A \sin(\phi)}{\sqrt{1 + \omega^2 \tau^2}} + C \text{ so that } C = \frac{A \sin(\phi)}{\sqrt{1 + \omega^2 \tau^2}}$$

3. a. Prove that for an underdamped series R-L-C circuit with  $V(t) = \sin \omega_0 t$  and  $v_c = dv_c/dt = 0$  at  $t = 0$  the solution for the capacitor voltage is:

$$v_c(t) = Q \left[ e^{-\alpha t} \left( \cos \beta t + \frac{\alpha}{\beta} \sin \beta t \right) - \cos \omega_0 t \right], \text{ where } Q = \frac{1}{R} \sqrt{\frac{L}{C}} = \frac{\omega_0}{2\alpha}$$

Thus as expected from the results in Chapter 4 for a band pass filter, the steady state amplitude of these oscillations is  $Q$  (the quality factor for the filter). Note that for a high  $Q$  system, the damping is minimal, so that  $\alpha \ll \omega_0$  and  $\beta \approx \omega_0$ , so that

$$v_c(t) \approx -Q(1 - e^{-\alpha t}) \cos \omega_0 t,$$

Once again, we use the fact that the particular (steady state) solution is the solution obtained from circuit theory, where

$$\underline{V}_C = \underline{V} / (1 + j\omega RC - \omega^2 LC)$$

so that at resonance ( $\omega_0 = (LC)^{-1/2}$ )

$$\underline{V}_C = \underline{V} / (j\omega_0 RC) = -j\underline{V}Q \text{ with } Q = (L/C)^{1/2} \cdot (1/R)$$

Hence for  $v(t) = \sin \omega_0 t$ ,

$$V_{CP}(t) = Q \sin(\omega_0 t - .5\pi) = -Q \cos \omega_0 t.$$

Next, as shown in the notes, the solution to the homogenous equation (Eq. 5.25) is

$$V_{CH}(t) = e^{-\alpha t} (A \cos \beta t + B \sin \beta t), \text{ so that}$$

$$(1) V_C(t) = e^{-\alpha t} (A \cos \beta t + B \sin \beta t) - Q \cos \omega_0 t.$$

To find  $A$  and  $B$  we use the conditions that at  $t = 0$ ,  $v_c(0) = 0$  and  $dv_c/dt = 0$ . Thus we have

$$V_C(0) = A - Q = 0, \text{ so that } A = Q.$$

Also

$$(dv_c/dt)_{t=0} = -\alpha A + \beta B = 0, \text{ so that } B = \alpha A / \beta = \alpha Q / \beta$$

Substituting these values into (1) we find

$$v_c(t) = Q \left[ e^{-\alpha t} \left( \cos \beta t + \frac{\alpha}{\beta} \sin \beta t \right) - \cos \omega_0 t \right]$$

For a high  $Q$  system,  $(\alpha/\beta) \ll 1$  and  $\beta \approx \omega_0$ , so that  $v_c(t)$  reduces to

$$v_c(t) = Q \cos \omega_0 t [e^{-\alpha t} - 1], \text{ the result given in the problem statement.}$$

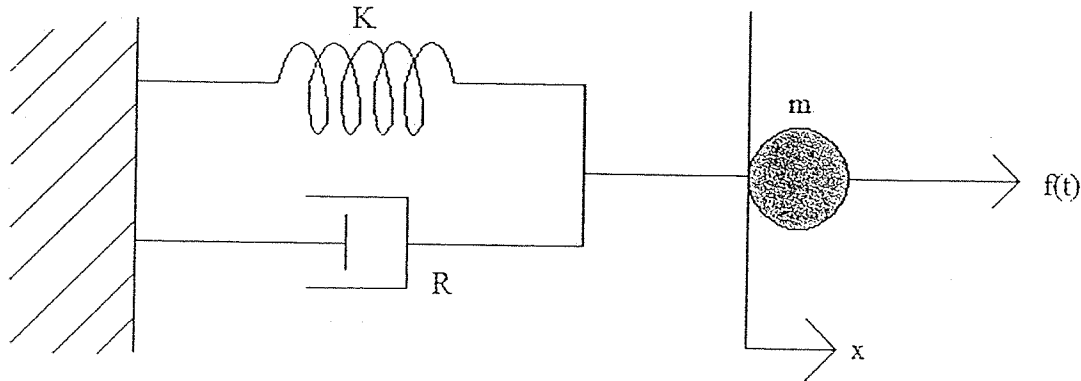
b. Show that for a high  $Q$  system it takes about  $Q$  cycles before the full amplitude of the steady state solution is achieved. Hint: show that after  $Q$  cycles,  $\alpha t = \pi$ , so that  $e^{-\alpha t}$  becomes negligible ( $e^{-\pi} = .0432$ ).

As shown in the notes,  $\alpha = \omega_0 / 2Q$ . Hence  $e^{-\alpha t} = \exp -\omega_0 t / 2Q$ . Thus the exponential falls to  $e^{-\pi}$ , when  $\pi = \omega_0 t / 2Q$ , or  $t = 2\pi Q / \omega_0 = Q / f_0 = QT$ , where  $T$  is the period of the oscillations.

6. The governing equation for the mechanical system with a mass, spring and dashpot (damper) shown below has the form

$$m\ddot{x} + R\dot{x} + \kappa x = f(t)$$

where  $m$  is the mass (kg),  $R$  is the viscous resistance (N-s/m),  $\kappa$  is the spring constant (N/m), and  $f$  is the force (N).



When a force of 1 N is applied, the displacement is found undergo about 20 oscillations in 20 s before “reaching” an equilibrium displacement of 1 mm. (By reaching equilibrium we mean the amplitude of the oscillations drops to about  $e^{-\pi}$  compared to the amplitude of the first oscillation.) Show that  $m \approx 25.3$  kg,  $R \approx 7.9$  N-s/m, and  $\kappa \approx 10^3$  N/m.

**Hint:** A system that takes 20 oscillations to damp out has  $\beta \gg \alpha$ , so that  $\omega_0 \approx \beta$ .

**From the data given,**

$$x_{Eq} = 10^{-3} \text{ m}; Q = 20; \text{ and } T = 1/f = 2\pi/\omega_0 = 1 \text{ s, so that } \omega_0 = 2\pi.$$

To find  $m$ ,  $R$ , and  $\kappa$ , note first that

$$\text{At equilibrium, } \kappa = 1/x_{Eq} = 10^3 \text{ N/m}$$

Dividing the governing equation by  $m$ , and comparing the homogenous equation (when  $f = 0$ ) with Eq. 5.22, we see that  $\kappa/m = \omega_0^2$  so that  $\underline{m =}$

$$\underline{\kappa/\omega_0^2 = 10^3/4\pi^2 = 25.3 \text{ kg.}}$$

From the same equation we also see that

$$R/m = \omega_0/Q, \text{ so that } R = m\omega_0/Q = 25.3 \cdot 2\pi/20 = \underline{7.9 \text{ N-s/m}}$$