The Univeristy of Derby
Faculty of Arts, Design and Technology

Efficient Acoustic Modelling of Large Spaces using Time Domain Methods

Analysis of Time Domain Numerical Methods for Acoustic Modelling of Large Spaces

Simon Durbridge

April 13, 2017

Submitted for in part-fulfilment of the requirements for the MSc in Audio Engineering.

for Bethany

Acknowledgements

I would like to dedicate this work to anyone of remote importance.

Contents

Acronyms

Use the template *acronym.tex* together with the Springer document class SVMono (monograph-type books) or SVMult (edited books) to style your list(s) of abbreviations or symbols in the Springer layout.

Lists of abbreviations, symbols and the like are easily formatted with the help of the Springer-enhanced `description` environment.

ABC Spelled-out abbreviation and definition
BABI Spelled-out abbreviation and definition
CABR Spelled-out abbreviation and definition

Chapter 1
Introduction

The intro Text

1.1 Context

1.2 Problem Definition

Real time acoustic modelling could be of significant benefit to many applications; Engineers could make design changes and see results ’on the fly’, and entertainment users could have more realistic experiences. These benefits should be possible for an arbitrary number of sources and receivers, in proportionally large environments with high quality results. Is it possible to further reduce computation time for simulations of large acoustic problems, to provide results in real time for the full human audio frequency range? There are two ’branches’ of computation solution that should be considered: the direct solution i.e. direct outputs or audio samples from the simulation, and indirect solutions i.e. a system impulse response that may be convolved with mixed source signals in order to create an auralization of the system.//

Fig. 1.1 A visualisation of a 2D explicit FDTD simulation [**?**]

Chapter 2
Loudspeaker Systems & Large Room Acoustics

Abstract Acoustics is a branch of physics that aims to characterise Newton's law of motion applied to wave propagation, while obeying the physical conservation law and often focussing on propagation in an audible spectrum. This characterisation of sound propagation is intrinsically linked to many other branches of physics, as well as psychoacoustics and perception. Many aspects of acoustic modelling may be of interest when considering the design and application of loudspeaker systems. Both small and large scale simulations may allow a user to make informed decisions about the design and placement of a loudspeaker system, so that the performance of the system may be validated and optimised before application. In this chapter we will evaluate the lossless acoustic wave equation for gasses, and consider the application of the wave equation in bounded space. We will then consider some specific use cases for applying such an equation for modelling loudspeaker system performance.

2.1 The Acoustic Wave Equation

In the Mcgraw-Hill Electronic and Electrical Engineering Series of books, the late L Beranek authored the Acoustics volume. This volume contains an elegant summary of the wave equation, that will be the subject of paraphrase in the following section.

2.1.1 The Wave Equations

Acoustic waves are classified as fluctuations of pressure in a given medium. In room acoustics and loudspeaker system engineering, these fluctuations are often cyclical in nature around an ambient pressure, as opposed to the jets described in aeroacoustic study. Similar to the behaviour of heat convection or fluid diffusion, these cyclical fluctuations propagate and spread through the medium of interest. As these fluctuations of pressure propagate energy is often lost, and eventually the medium will often come to a state of relative rest where the energy of the propagating waves have been almost entirely dissipated. It is possible to calculate an approximate solution the the propagation of pressure through a space, by solving a system of second order partial differential equations that can be collectively lumped into a 'Wave Equation'. Below, we will introduce the three building blocks of the wave equation in both one dimension, and three dimensions (based on vector notation). These building blocks are Newton's Second Law of Motion, the gas law, and the laws of conservation of mass.

To consider the wave equation, we should use the analogy of a small[1] volume of gas, within a larger homogeneous medium. The faces of the volume are frictionless, and only the pressure at any face impacts on the gas inside the volume.

One Dimension	Three Dimensions
Sound pressure p propagates across the medium like a plane wave, from one side to the other in the x direction at a rate equal to the change in space $\frac{\delta p}{\delta x}$	Sound pressure p propagates across the medium like a spherical wave, from one side to the other at a rate of $\mathbf{grad}\ p = \mathbf{i}\frac{\delta p}{\delta x} + \mathbf{j}\frac{\delta p}{\delta y} + \mathbf{k}\frac{\delta p}{\delta z}$ where i, j and k are unit vectors in the directions x, y and z.
Force acting on the volume in the positive x direction can thus be described as $-(\frac{\delta p}{\delta x}\Delta x)\Delta y\Delta z$	Force acting on the volume in the positive x direction can thus be described as $-[i(\frac{\delta p}{\delta x}\Delta x)\Delta y\Delta z) + j(\frac{\delta p}{\delta y}\Delta y)\Delta x\Delta z) + k(\frac{\delta p}{\delta z}\Delta z)\Delta x\Delta y)]$
A positive gradient causes acceleration in the $-x$ direction	$\leftarrow$
Force per unit volume is given by dividing both sides of the previous equation by the volume V, $\frac{f}{V} = -\frac{\delta p}{\delta x}$	Force per unit volume is given by dividing both sides of the previous equation by the volume V, $\frac{f}{V} = -\mathbf{grad}p$
Newton's second law of motion dictates that the rate of change of momentum in the volume must balance with force per unit volume, and we can assume the mass of gas in the volume is constant. The force mass balance can be described as $\frac{f}{V} = -\frac{\delta p}{\delta x} = \frac{M}{M}\frac{\delta u}{\delta t} = \rho'\frac{\delta u}{\delta t}$	$\leftarrow$ The force mass balance can be described as $\frac{f}{V} = -\mathbf{grad}p = \frac{M}{M}\frac{Dq}{Dt} = \rho'\frac{Dq}{Dt}$

[1] rectilinear

u is the velocity of gas in the volume, ρ' is the density of the gas, and $M = \rho' V$ is the mass of gas in the volume.	where q is the vector velocity, ρ' is the density of gas in the volume, $M = \rho' V$ is the total mass of gas in the volume. $\frac{D}{Dt}$ represents the total rate of change of velocity of a section of gas in the volume, and can be composed as $\frac{Dq}{Dt} = \frac{\delta q}{\delta t} + q_x \frac{\delta q}{\delta x} + q_y \frac{\delta q}{\delta y} + q_z \frac{\delta q}{\delta z}$ where q_x, q_y and q_z are the components of the particle velocity q in each direction. As this is a linear wave equation approximation, these velocity components have no cross terms.
If the change in density of gas in the volume is sufficiently small, the ρ' will be approximately equal to the average density ρ_0, thus simplifying the equations above to $-\frac{\delta p}{\delta x} = \rho_0 \frac{\delta u}{\delta t}$	If the particle velocity vector is sufficiently small, the change of momentum of the gas is approximately the same as the momentum of the volume at any arbitrary point, and the density of gas within the volume ρ' will be approximately equal to the average density ρ_0. Thus the above can be written as $-grad\, p = \rho_0 \frac{\delta q}{\delta t}$
This kind of approximation may be appropriate as long as the maximum pressure is appropriately low, so that the behaviour of the air is linear, often quoted to be at or under the threshold of pain for human hearing or 120dB SIL.	←
→	Assuming that the gas of the volume is ideal, then the gas law $PV = RT$ should hold true. Here, T is the temperature in degrees Kelvin, and R is a constant based on the mass of the gas. For this approximation we assume that the system is adiabatic, and that T and R are lumped into a gas constant which for air is $\gamma = 1.4$.
In differential form, the relationship between pressure and volume for an adiabatic expansion the volume is $\frac{dP}{P} = \frac{-\gamma dV}{V}$ i.e. changes in pressure scale with changes in volume by this γ value.	←

→	If perturbations in pressure and volume due to a sound wave, p for pressure and τ for volume respectively, are sufficiently small compared to the rest values P_0 and V_0; the time based derivative of the above equation can be written as follows: $\frac{1}{P_0}\frac{\delta p}{\delta t} = \frac{-\gamma}{V_0}\frac{\delta \tau}{\delta t}$
As the wave equation being derived is concerned with the transport of pressure within a volume, a continuity expression must be applied. The conservation of mass states that the total mass of gas in the volume must remain constant. This conservation law brings a unique relationship between discrete velocities at the boundary of the volume:	←
If the volume is displaced by some rate ε_x, air particles at either boundary of the volume must be displaced at an equal rate for the mass of the volume to remain constant. As such if the left side of the volume is displaced with a velocity, in a given time step the particles at the right hand boundary must also be displaced. This can be written as $\varepsilon_x + \frac{\delta \varepsilon_x}{\delta x}\Delta x$ The difference between this velocity and a subsequent change in volume τ multiplied by the volume gives $\tau = V_0 \frac{\delta \varepsilon_x}{\delta x}$.	If the mass of gas within the box must remain constant, the vector displacement will directly change the volume by some rate, as the two must balance to satisfy the continuity equation. This can be written as $\tau = V_0\ div\ \varepsilon$
Differentiating this with respect to time gives: $\frac{\delta \tau}{\delta t} = V_0 \frac{\delta u}{\delta x}$ where u is the instantaneous particle velocity	Differentiating this with respect to time gives: $\frac{\delta \tau}{\delta t} = V_0\ div\ q$ where q is the instantaneous particle velocity
The one dimensional wave equation in rectangular coordinates can be created by combining the above statements about the equation of motion, the gas law and the continuity equation. The combination of the gas law and continuity equation gives $\frac{\delta p}{\delta t} = -\gamma P_0 \frac{\delta u}{\delta x}$	The three dimensional wave equation in rectangular coordinates can be created by combining the above statements about the equation of motion, the gas law and the continuity equation. The combination of the gas law and continuity equation gives $\frac{\delta p}{\delta t} = -\gamma P_0 div\mathbf{q}$

When differentiated with respect to time, this gives: $\frac{\delta^2 p}{\delta t^2} = -\gamma P_0 \frac{\delta^2 u}{\delta t \delta x}$	When differentiated with respect to time this gives: $\frac{\delta^2 p}{\delta t^2} = -\gamma P_0 div \frac{\delta q}{\delta t}$
Differentiating the momentum equation derived above with respect to time gives $-\frac{\delta^2 p}{\delta t^2} = \rho_0 \frac{\delta^2 u}{\delta x \delta t}$	The divergence of the momentum equation derived above gives: $-div = \rho_0 div \frac{\delta q}{\delta t}$ Replacing the divergence $(gradp)$ term with the Lapacian operator$\nabla^2 p$ produces $-\nabla^2 p = \rho_0 div \frac{\delta^2 p}{\delta t}$
Combining the above equations gives: $\frac{\delta^2 p}{\delta x^2} = \frac{\rho_0}{\gamma P_0} \frac{\delta^2 p}{\delta t^2}$	Combining the above equations gives: $\nabla^2 p = \frac{\rho_0}{\gamma P_0} \frac{\delta^2 p}{\delta t^2}$
If we define c as the speed of propagation in the medium of interest, then $c^2 \approx \frac{\gamma P_0}{\rho_0}$ due to the fact that the speed of sound $c \approx (1.4 \frac{10^5}{1.18})^{\frac{1}{2}}$ where the ambient air pressure at sea level is $10^5 Pa$, 1.4 is the adiabatic constant γ (ratio of specific heats) for air, and ρ_0 is the density of air is approximately $1.8 kg/m^3$	$\leftarrow$
Finally we find that the 1 dimensional wave equation is: $\frac{\delta^2 p}{\delta x^2} = \frac{1}{c^2} \frac{\delta^2 p}{\delta t^2}$	Finally we find that the 3 dimensional wave equation is: $\nabla^2 p = \frac{1}{c^2} \frac{\delta^2 p}{\delta t^2}$ An explicit 3 dimensional expression of the pressure component of this equation is: $\nabla^2 p = \frac{\delta^2 p}{\delta x^2} + \frac{\delta^2 p}{\delta y^2} + \frac{\delta^2 p}{\delta z^2}$
This equation can also be expressed in terms of the instantaneous velocity in the volume as: $\frac{\delta^2 u}{\delta x^2} = \frac{1}{c^2} \frac{\delta^2 u}{\delta t^2}$	This equation can be expressed velocity vector $\nabla^2 q = \frac{1}{c^2} \frac{\delta^2 q}{\delta t^2}$ where $\nabla^2 q$ represents the gradient of pressure (velocity) in the volume.

In the above table we have derived wave equations, with forms of velocity and pressure as the independent variables. We have also shown that pressure, velocity, displacement and density are related within the system of equations, by differentiating and integrating with respect to space and time. As these forms of the wave equation are intrinsically coupled, it is possible to leverage this coupling when generating a numerical solution to the wave equation. It is also important to note that a significant number of assumptions have been taken when deriving these equations, and any solution to these equations may only be accurate when simulating a loss free, frictionless, homogeneous, ideal gas medium, where all perturbations are sufficiently small and fast that it is possible to reduce the complexity of the system.

2.2 Loudspeaker Systems

Instead of simply listing headings of different levels we recommend to let every heading be followed by at least a short passage of text. Furtheron please use the LaTeX automatism for all your cross-references and citations.

Methods

Difference Time Domain Method

Finite Difference Time Domain Method

Time Domain Method

Modelling Strategies

Work