

# Periodic Motion II.

## Review on SHM

mechanical system obeys equation of motion

$$\frac{d^2x}{dt^2} = -\omega^2 x, \quad f = \frac{\omega}{2\pi}, \quad T = \frac{2\pi}{\omega}$$

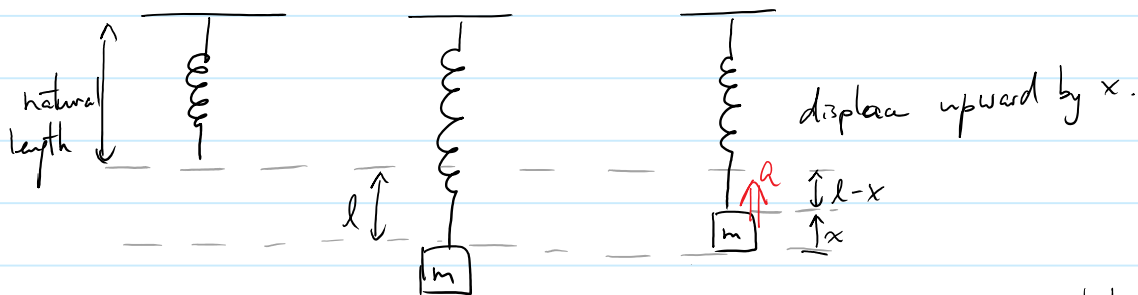
Solution:  $x(t) = A \cos(\omega t + \phi)$  where  $A, \phi$  are determined by initial condition  $(x_0, v_0)$

In general,  $x$  is a displacement from the equilibrium position  
can be linear or angular

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## Some examples of SHM

- Vertical spring-mass system



Force diagram for the mass  $m$ :

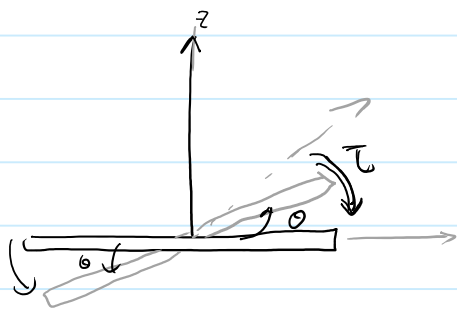
- At equilibrium: Upward force  $kl$ , downward force  $mg$ . Net force  $F = kl - mg = 0$ .
- At displacement  $x$ : Upward force  $k(l-x)$ , downward force  $mg$ .

Force equation for the displaced state:

$$F = ma$$
$$k(l-x) - mg = ma$$
$$\underbrace{kl - mg}_{=0} - kx = ma$$
$$\Rightarrow ma = -kx$$
$$\Rightarrow \ddot{x} = a = -\frac{k}{m}x$$

$\omega^2 = \frac{k}{m}$  same as horizontal spring. SHM

# Angular Oscillation



$\tau$  turning the rod back to original orientation.

$$\vec{\tau} \parallel -\hat{k}$$

$$\tau \propto \theta$$

$$\tau_z = -K\theta$$

c.f.  $F_{\text{spring}} = -kx$

$$\tau_z = I\alpha \Rightarrow \alpha = \frac{d^2\theta}{dt^2} = \frac{\tau}{I} = -\frac{K}{I}\theta$$

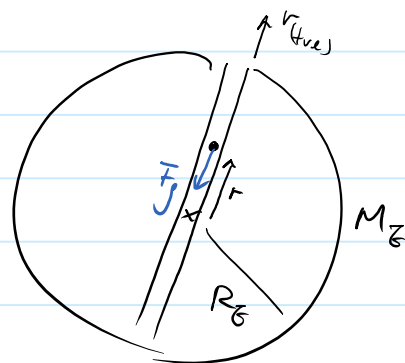
$$\omega^2 = \frac{K}{I}$$

$$\text{Period} = T = 2\pi \sqrt{\frac{I}{K}} \sim 2\pi \sqrt{\frac{\text{Inertia}}{\text{restoring strength}}} \quad \text{c.f. } T = 2\pi \sqrt{\frac{m}{k}}$$

- Tunnel through the Earth.

Recall the gravitational force inside a uniform solid sphere is

$$F_g = \frac{G M_E m}{R_E^3} \cdot r$$



A mass inside the tunnel through the Earth obeys.

$$\vec{F} = m\vec{a}$$

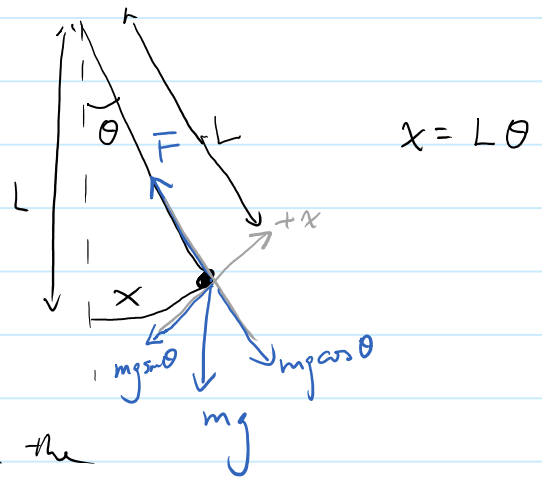
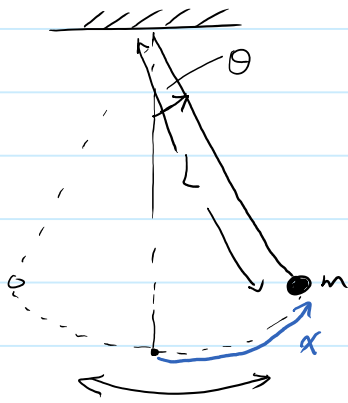
$$-\frac{G M_E m}{R_E^3} r = m \frac{dr}{dt^2}$$

$$\Rightarrow \frac{d^2 r}{dt^2} = - \left[ \frac{G M_E}{R_E^3} \right] \cdot r$$

$\downarrow \omega^2$

$$T = 2\pi \sqrt{\frac{R_E^3}{G M_E}} \sim 1.5 \text{ hr.}$$

# • Simple Pendulum



Let  $x$  be the arc length shown in the figure.

The acceleration along  $x$  is due to only the component of  $mg$  along  $x$ ; i.e.,  $mg \sin \theta$ .

$$F = ma$$

$$\Rightarrow -mg \sin \theta = m \frac{d^2 x}{dt^2}$$

$$x = L\theta \Rightarrow \frac{d^2 x}{dt^2} = L \frac{d^2 \theta}{dt^2}$$

$$\Rightarrow -g \sin \theta = L \frac{d^2 \theta}{dt^2} \quad \text{--- (*)}$$

for small  $\theta$  ( $\theta \lesssim 0.1$  rad or  $5.7^\circ$ )

$$\sin \theta = \theta - \frac{\theta^3}{3!} + \frac{\theta^5}{5!} - \dots$$

$$\approx \theta \quad \theta^3, \theta^5 \dots \text{ very small compare to } \theta.$$

(\*) becomes

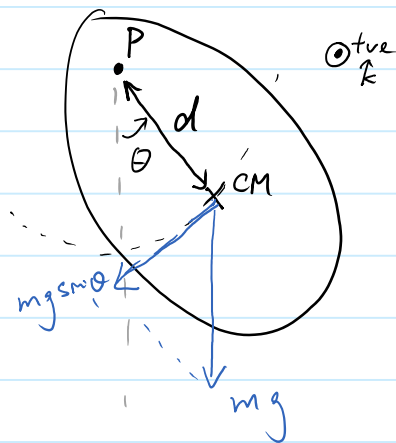
$$\frac{d^2 \theta}{dt^2} = -\frac{g}{L} \theta \quad \underline{\text{SHM}}$$

$$\omega^2 = \frac{g}{L}, \quad T = 2\pi \sqrt{\frac{L}{g}} \quad \text{independent of mass.}$$

## Physical Pendulum

Consider a rigid body pinned down on a wall at point P.

It can swing freely about P under gravity.



$$\vec{\tau} = -mg \sin \theta \cdot d \cdot \hat{k} = I_P \alpha \hat{k}$$

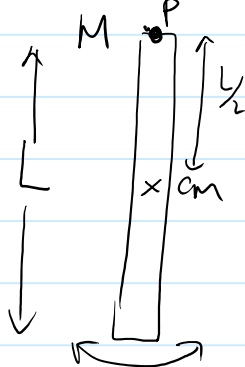
$$\Rightarrow I_P \alpha = -mgd \sin \theta$$

$$\alpha = \frac{d^2 \theta}{dt^2} = - \frac{mgd \sin \theta}{I_P} \approx - \frac{mgd}{I_P} \theta \quad \text{for small } \theta$$

$$\Rightarrow \boxed{\frac{d^2 \theta}{dt^2} = - \frac{mgd}{I_P} \theta} \quad \text{SIM.}$$

$$\omega^2 = \frac{mgd}{I_P}, \quad T = 2\pi \sqrt{\frac{I_P}{mgd}}$$

Example A rod swing about one of the ends

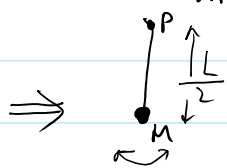
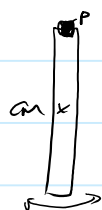


$$I_P = \frac{1}{3} ML^2$$

$$d = \frac{L}{2}$$

$$T = 2\pi \sqrt{\frac{\left(\frac{1}{3} ML^2\right)}{Mg \frac{L}{2}}} = 2\pi \sqrt{\frac{2}{3} \frac{L}{g}}$$

Can I treat the rod as a simple pendulum with all the mass concentrated at its CM?

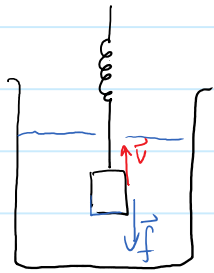


Simple pendulum

$$T = 2\pi \sqrt{\frac{\left(\frac{L}{2}\right)}{g}} = 2\pi \sqrt{\frac{L}{2g}} \neq 2\pi \sqrt{\frac{2}{3} \frac{L}{g}}$$

$\Rightarrow$  Ans: NO! shape matters!

# Damped Harmonic Oscillation



drag force:  $\vec{f} = -b\vec{v}$  opposite to  $\vec{v}$ .

$$\vec{F}_{\text{net}} = \vec{F}_{\text{spring}} + \vec{f} = m\vec{a}$$

$$\Rightarrow -bv - kx = ma$$

$$\Rightarrow \boxed{\frac{d^2x}{dt^2} + \frac{b}{m} \frac{dx}{dt} + \frac{k}{m} x = 0}$$

$v = \frac{dx}{dt}$   
eq<sup>n</sup> of motion.

Solution:  $x(t) = A e^{-\frac{b}{2m}t} \cos(\omega't + \phi)$

where  $\omega' = \sqrt{\frac{k}{m} - \frac{b^2}{4m^2}} = \frac{1}{2m} \sqrt{(2\sqrt{km})^2 - b^2}$

new angular frequency  $\omega'$  (indicated by a red arrow from the text to the  $\omega'$  term in the equation)

old angular frequency  $\sqrt{\frac{k}{m}} = \omega_0$  (indicated by a red arrow from the text to the  $2\sqrt{km}$  term in the equation)

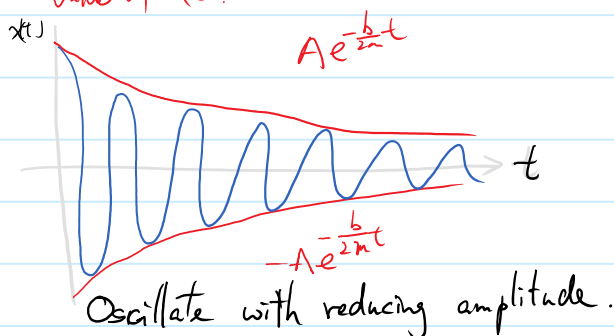
Practice: Try to verify  $x(t)$  is the solution of the eq<sup>n</sup> of motion.

Different values of  $k$ ,  $m$  &  $b$  result in different  $\omega'$  and different types of motion.

Case 1 (underdamp)

$$b < 2\sqrt{km} \Rightarrow \omega' > 0$$

$$x(t) = \underbrace{A e^{-\frac{b}{2m}t}}_{\text{govern the maximum value of } x(t)} \underbrace{\cos(\omega't + \phi)}_{\text{oscillate between } (-1, 1)} = A(t) \cos(\omega't + \phi)$$



Case 2 (critical damp)

$$b = 2\sqrt{km} \Rightarrow \omega' = 0$$

$\Rightarrow$  No oscillations!

$$x(t) = A e^{-\frac{b}{2m}t} \cos(\phi) \propto e^{-\frac{b}{2m}t}$$

Case 3 (overdamp)

$$b > 2\sqrt{km} \Rightarrow \omega' \text{ is complex number!}$$

$\Rightarrow$  No oscillations, too!

Energy

$$E = \frac{1}{2}mv^2 + \frac{1}{2}kx^2$$

$$\frac{dE}{dt} = \frac{1}{2}m(2v)\dot{v} + \frac{1}{2}k(2x)\dot{x}$$

$$= mv a + kxv$$

$$= (ma + kx)v$$

but  $ma + kx + bv = 0$

$$= -bv \cdot v \leftarrow \text{negative, energy loss.}$$

$$= \vec{f} \cdot \vec{v}$$

$$= \text{Power by drag force}$$

Forced Oscillation

Consider to add a periodic driving force

$$F(t) = F_{\max} \cos(\omega_d t) \quad \omega_d \text{ is the driving freq.}$$

$$\text{Eqn of motion: } ma = -kx - bv + F(t)$$

Long time behaviour.  $x(t) \approx A(\omega_d) \cos(\omega_d t + \phi)$   
(for very large  $t$ )

the object oscillates with the driving freq. at an  $\omega$ -dependent amplitude.

$$A(\omega_d) = \frac{F_{\max}}{\sqrt{(k - m\omega_d^2)^2 + b^2\omega_d^2}} = \frac{F_{\max}}{m} \cdot \frac{1}{\sqrt{(\omega_0^2 - \omega_d^2)^2 + \frac{b^2}{m}\omega_d^2}}$$

$A(\omega_d)$  is maximum when  $\omega_d \approx \omega_0$ .

$\nearrow$  goes to zero

$\nwarrow$  This characteristic is called resonance.

