

# Computational Assemblies: Analysis, Design, and Fabrication

Peng Song

Ziqi Wang

Marco Livesu



# Timetable

		Peng	Ziqi	Marco
Introduction	~20 mins	X		
Computational analysis of assemblies	~ 50 mins	X		
Computational design of assemblies	~50 mins		X	
Computational fabrication of assemblies	~ 50 mins			X
Q & A	~ 10 mins	X	X	X

# Computational Design of Assemblies

- Our goal is to design assemblies to achieve required objectives with the help of computational methods.



Puzzle



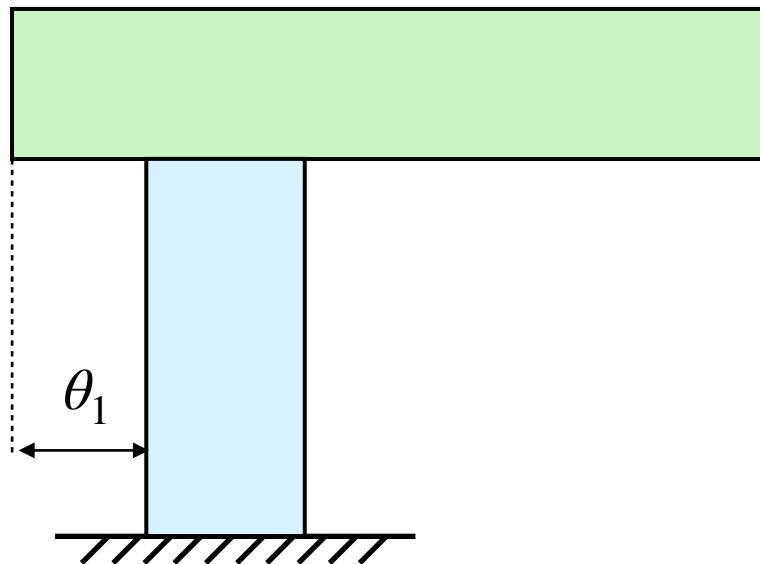
Furniture



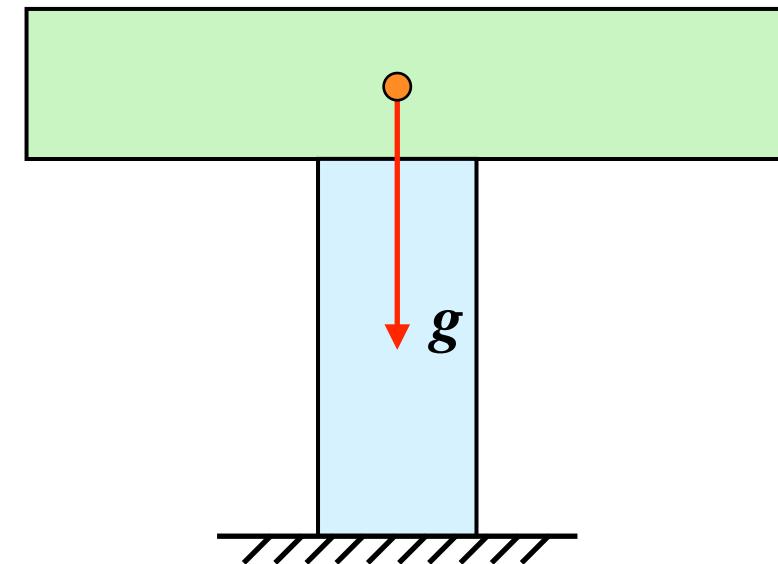
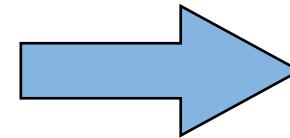
Architecture

# Case Study

- Two main components: **parts' geometry and design objectives.**



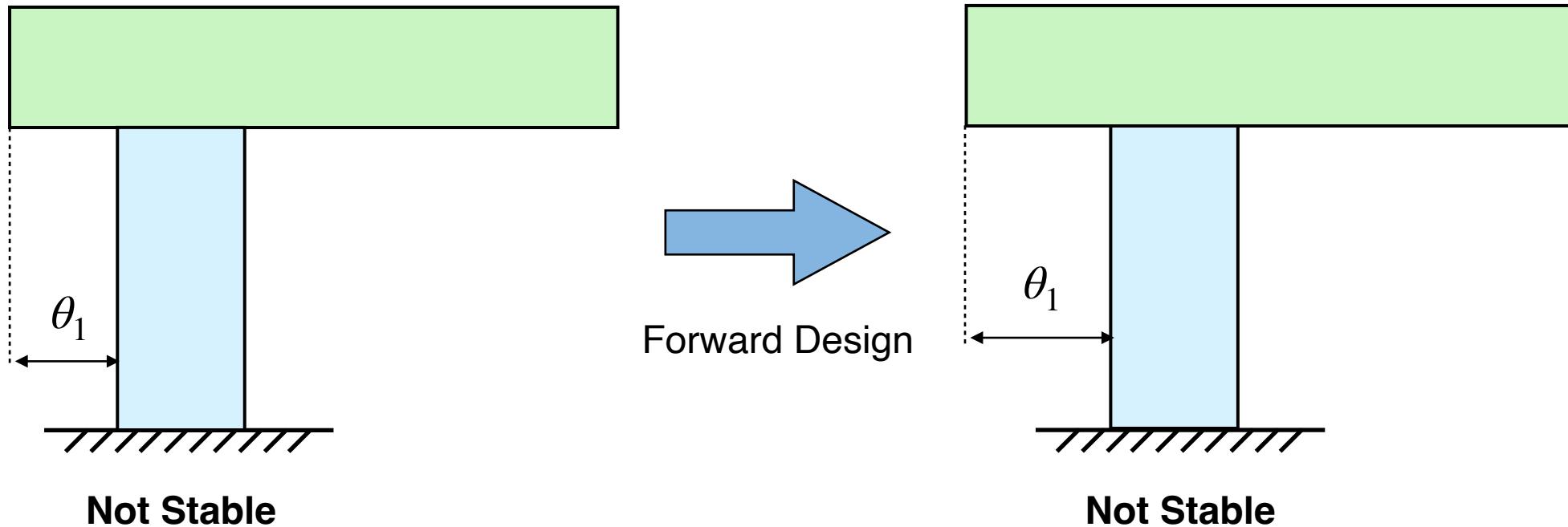
Parametric Model ( $t$ )



Equilibrium under Gravity

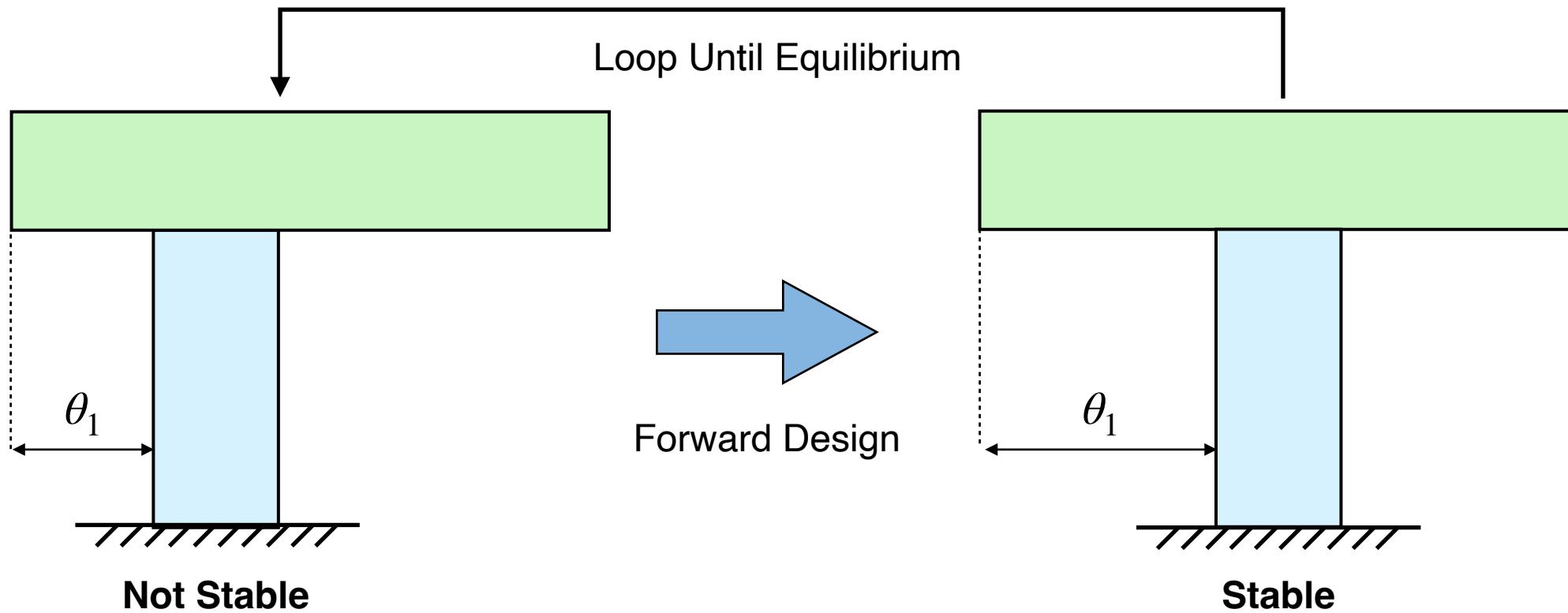
# Forward Design Framework

- Manually tune the design parameters until the assembly can stay in equilibrium under gravity.



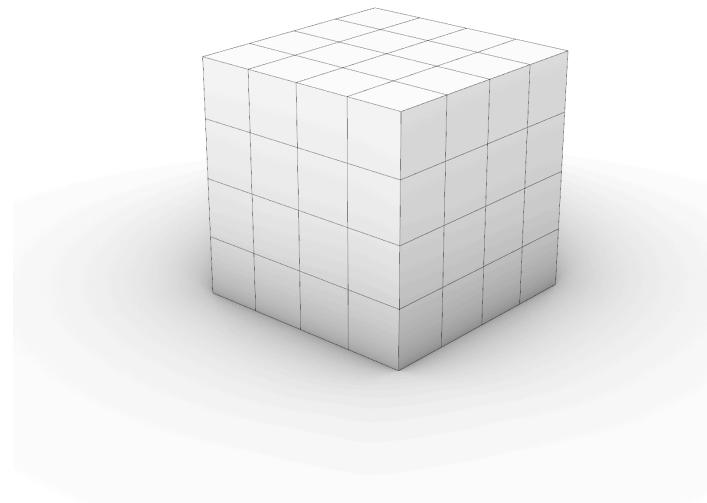
# Forward Design Framework

- More design iterations are required if the current design is not stable.



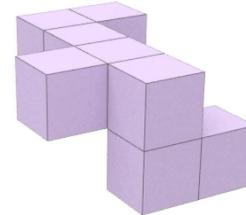
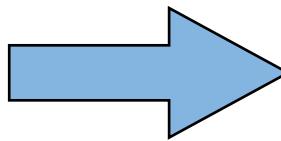
# Challenges of Forward Design

- The problem can have an enormous design space but finding one feasible solution is non-trivial.



**Design Space**

$$6^{64}$$

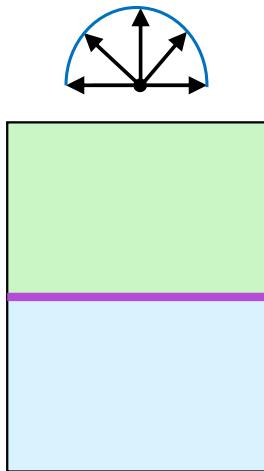
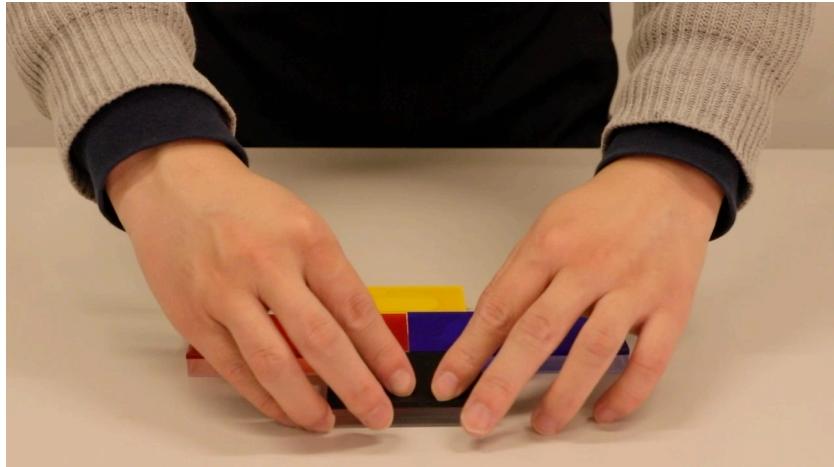


**Objectives**

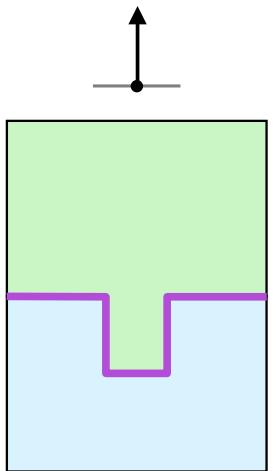
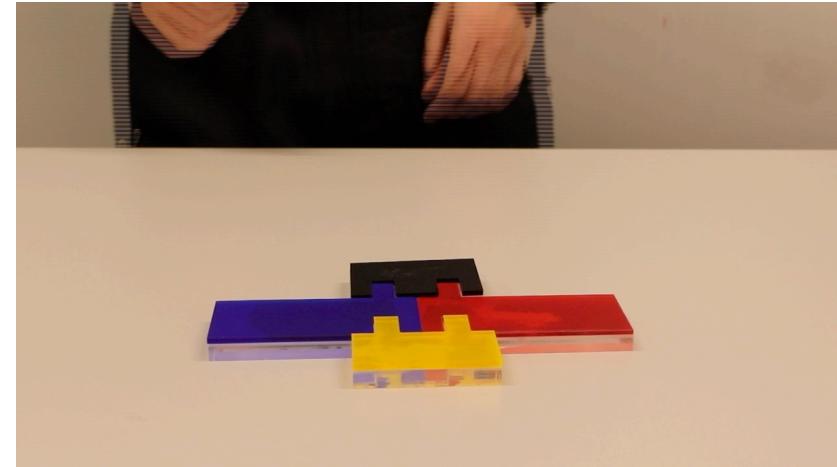
Stable under arbitrary forces

# Challenges of Forward Design

- The problem can have multiple design objectives.
- They might be contradicting.



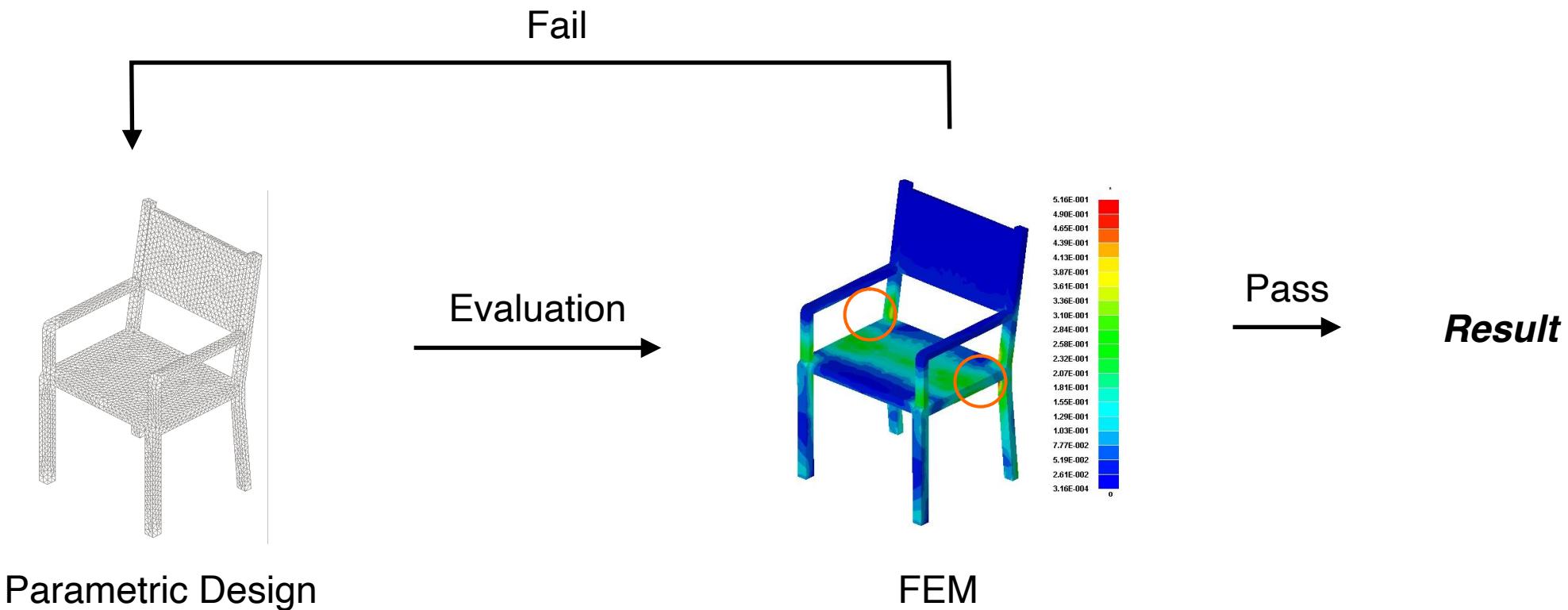
Easy-to-Assemble  
Non-stable



Hard-to-Assemble  
Stable

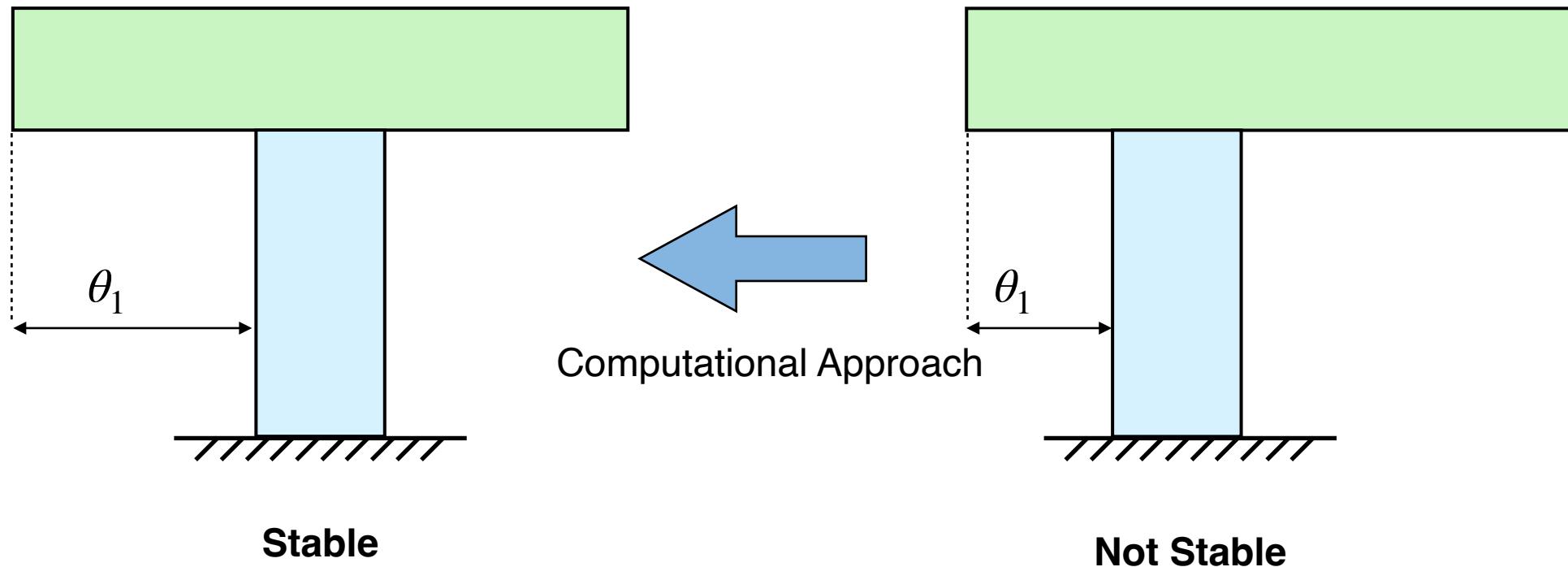
# Challenges of Forward Design

- Some evaluation processes are not time-efficient.



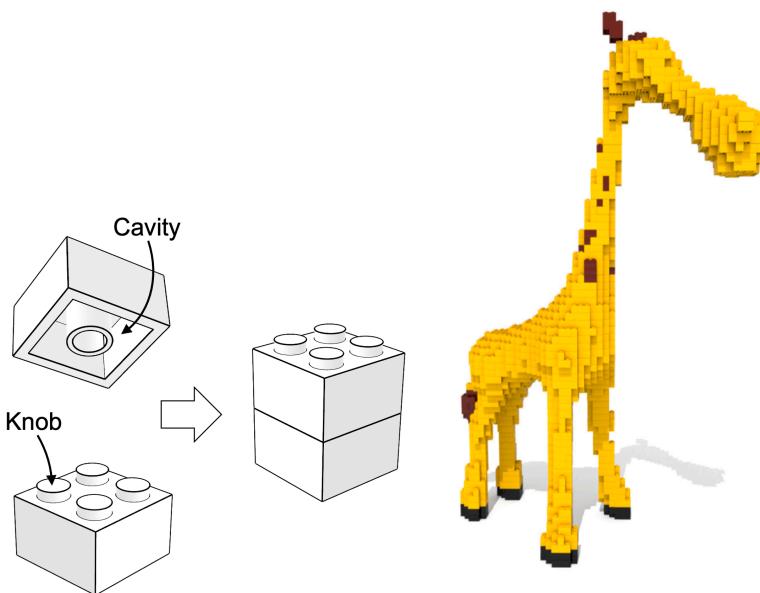
# Inverse Design Framework

- Automatically generate assemblies that satisfy design objectives.



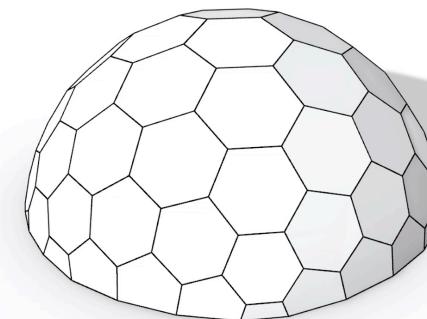
# Part Geometry

- Discrete Geometry: searching algorithm.
- Continuous Geometry: gradient-based algorithm.



Discrete Geometry

[Luo et al. 2015]

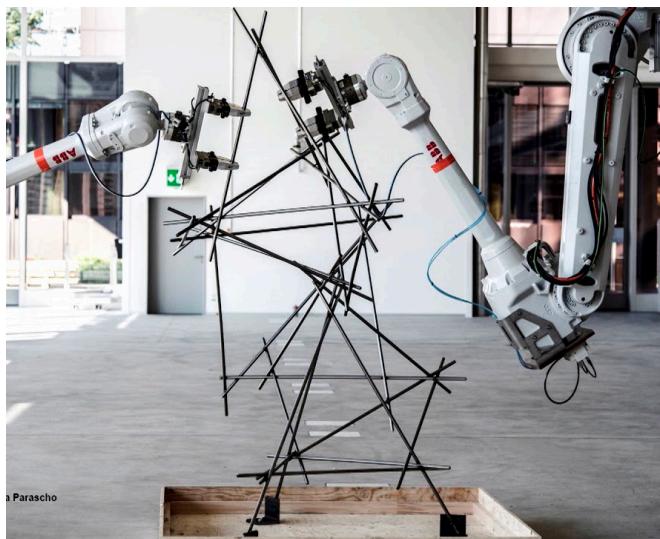


Continuous Geometry

[Wang et al. 2019]

# Objectives

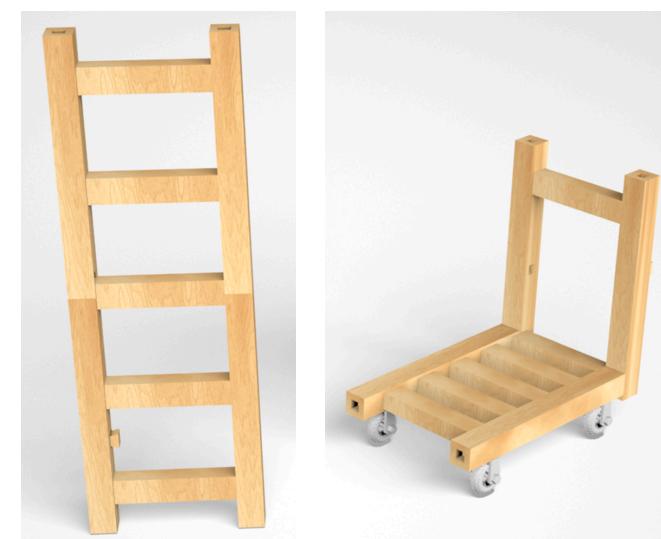
**Assemblability**



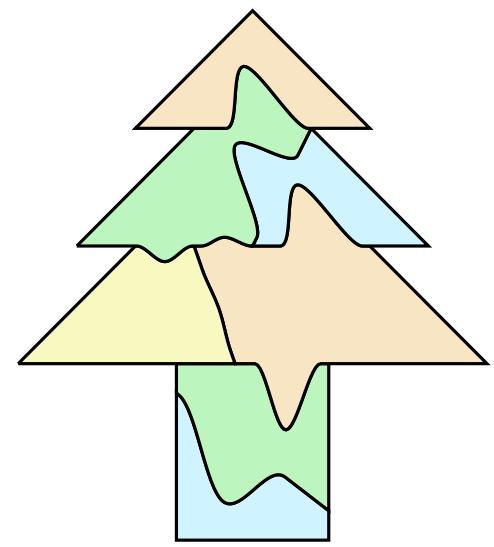
**Fabricability**



**Functionality**



**Stability**



[Parascho et al. 2017]

[Cignoni et al. 2014]

[Song et al. 2017]

[Wang et al. 2021]

# Stability Optimization

- Stability is the most fundamental requirement.
- Designing stable structures without glue/mortar is non-trivial.



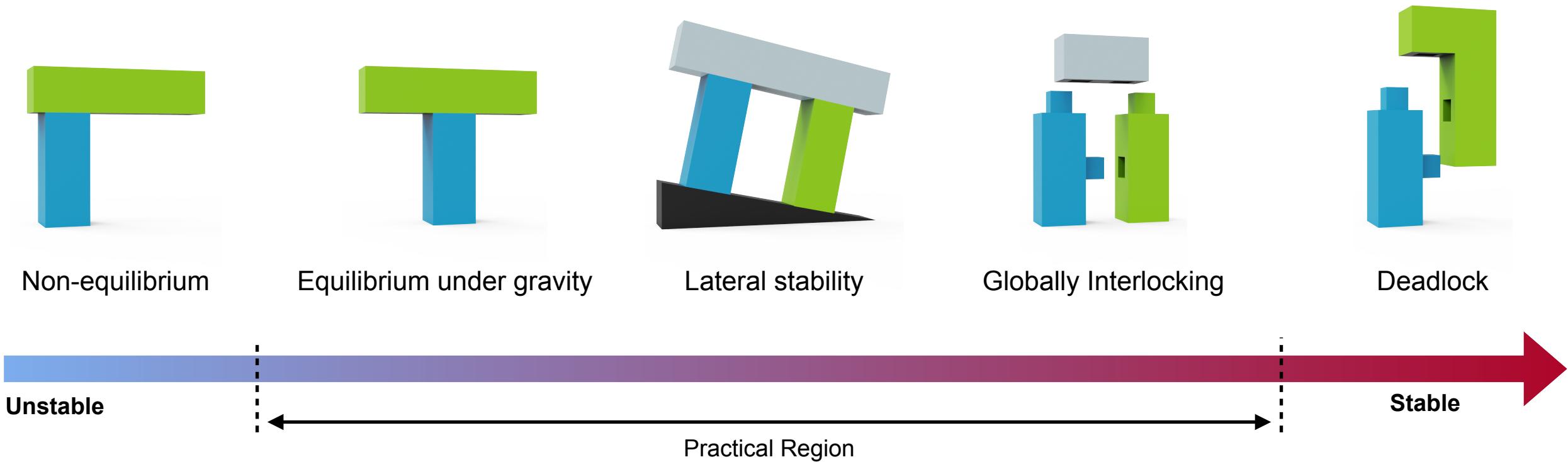
[Nara Todaiji]



[MIT Sean Collier Memorial]

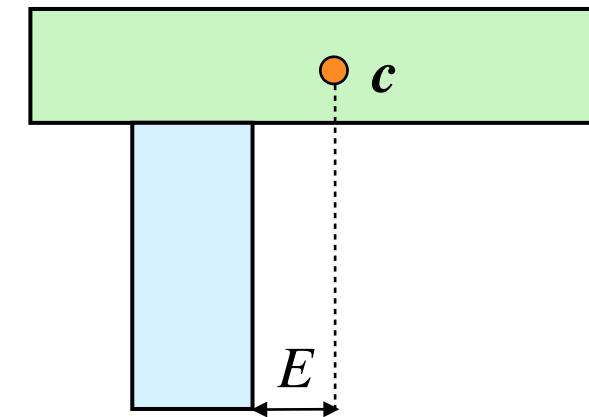
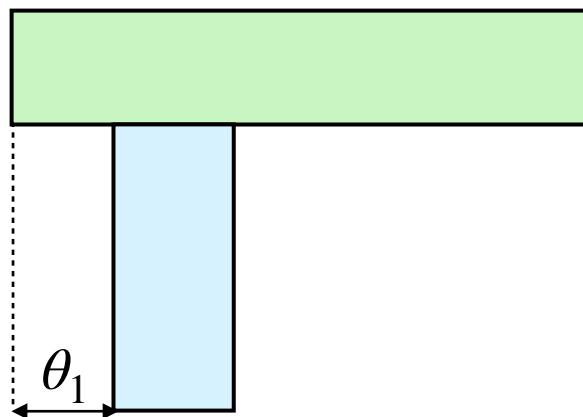
# Stability Spectrum

- Besides gravitational equilibrium, we will also cover other types of structural stability.



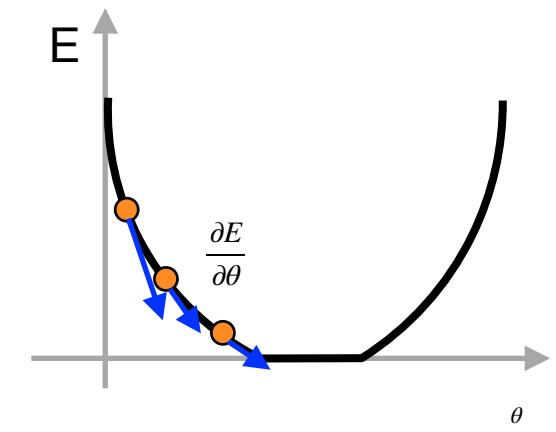
# Overview

- Part 1: General stability optimization framework using the gradient information.



$$\theta \xrightarrow{\partial} c \xrightarrow{\partial} E$$

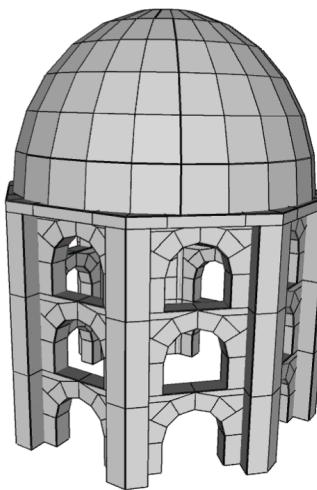
Step 3  
Sensitivity Analysis



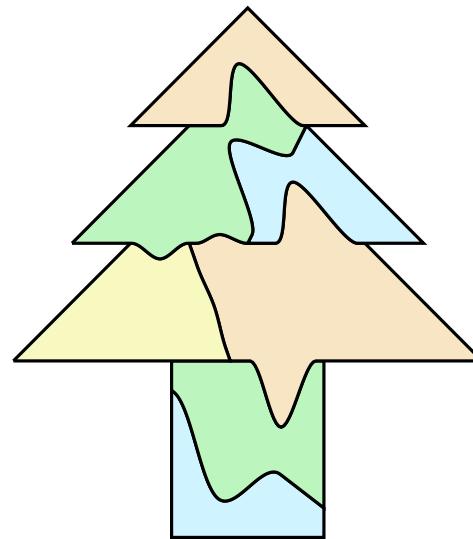
Step 4  
Numerical Optimization

# Overview

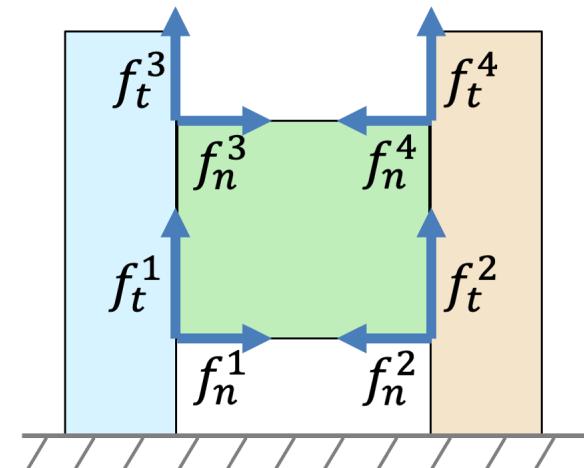
- Part 2: Stability optimization for gravitational equilibrium.
  - Force-based equilibrium method
  - Kinematic-based equilibrium method
  - Friction



[Whiting et al 2009, 2012]



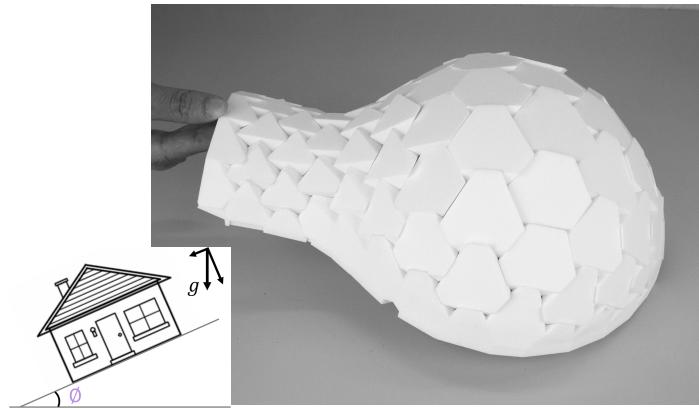
[Wang et al. 2021]



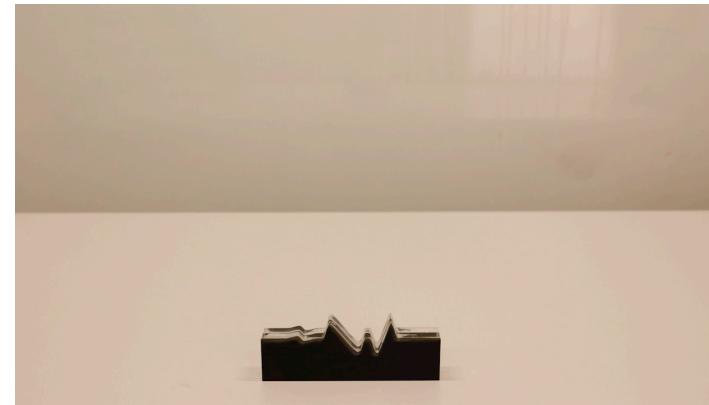
[Yao et al. 2017]

# Overview

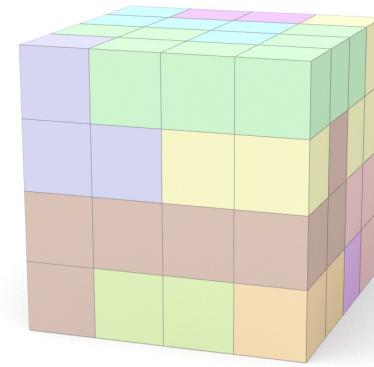
- Part 3: Design for stability under other types of forces
  - Lateral stability
  - Scaffolding-free assembly
  - Globally interlocking



[Wang et al. 2019]



[Wang et al. 2021]



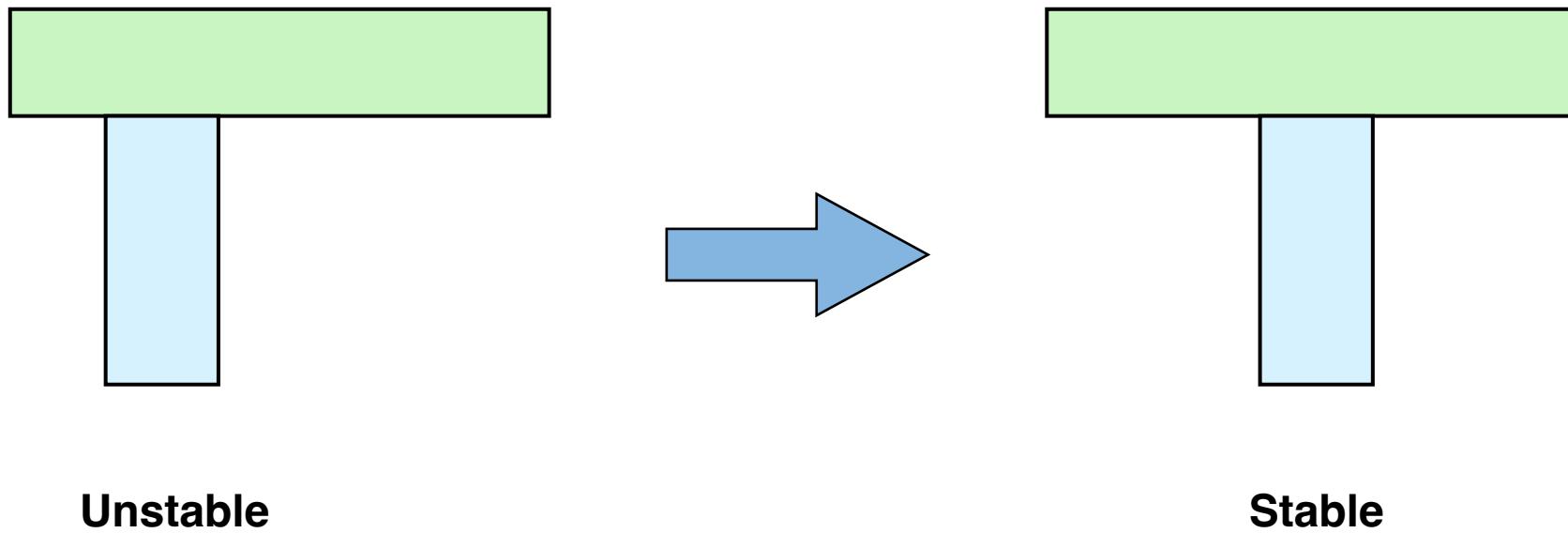
[Wang et al. 2018]

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# Part 1: General Stability Optimization Framework

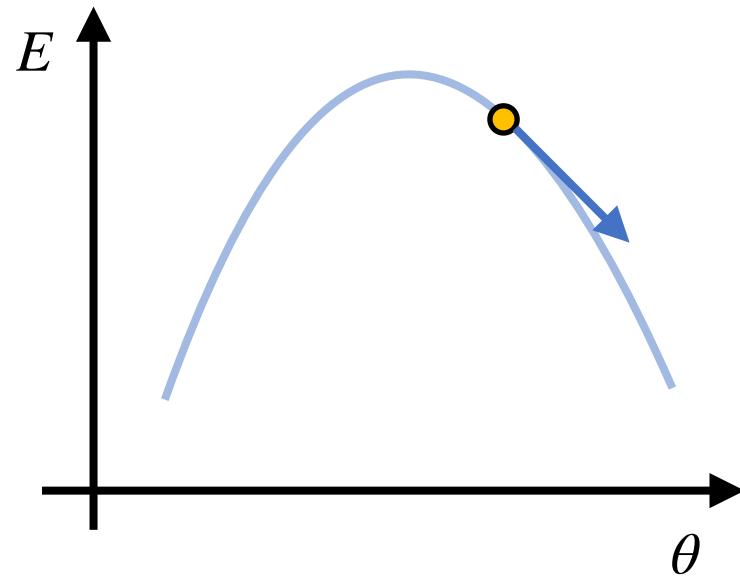
# Stability Optimization

- By alternating the parts' geometry, making the assembly stable under certain loading conditions.



# Gradient-based Optimization

- Gradient-based optimization is the most common approach to solve the inverse design problem.



**Strategy #1:** Take a random downhill slope.

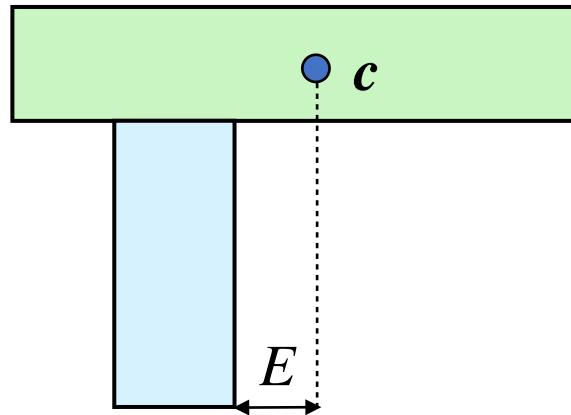
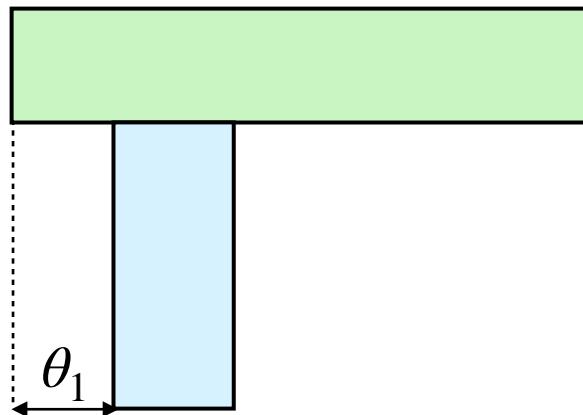
**Slow**

**Strategy #2:** Take the *steepest slope!*

**Fast**

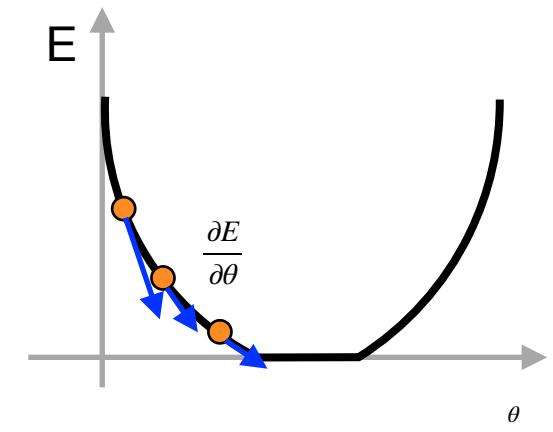
# Gradient-based Stability Optimization

- Gradient-based stability optimization has four main components:



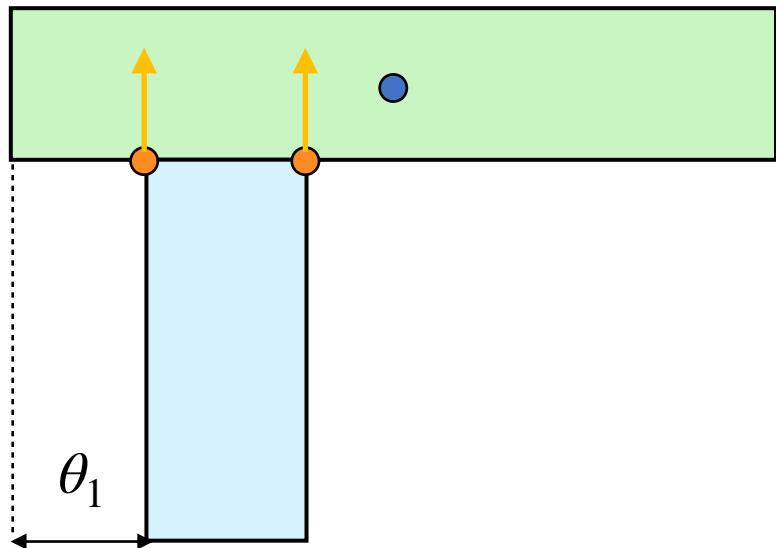
$$\theta \xrightarrow{\partial} c \xrightarrow{\partial} E$$

Step 3  
Sensitivity Analysis



# Step #1 Geometrical Property

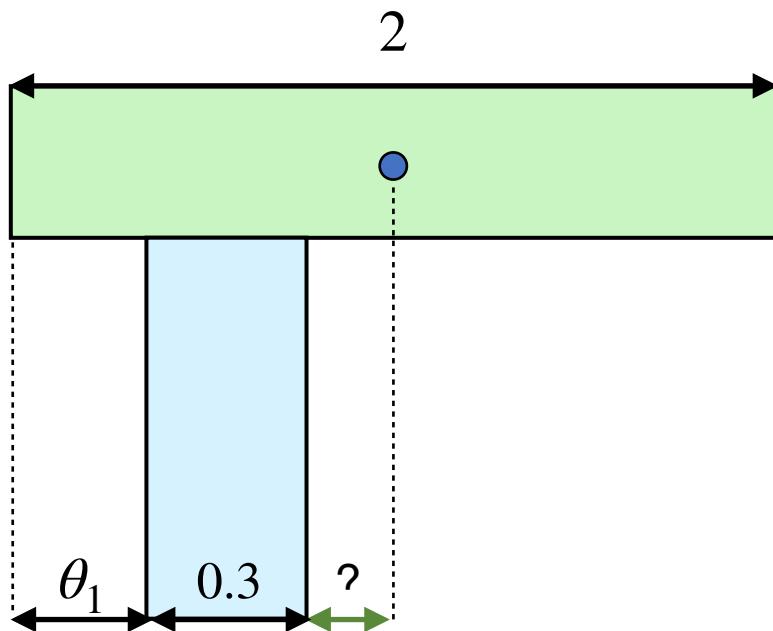
- Compute necessary geometrical properties for stability analysis.



1. Contact Points
2. Contact Normals
3. Parts' Centroids
4. Parts' Volumes

# Step #2 Infeasibility Measurement

- Compute the infeasibility energy, which measures how unstable the structure is.

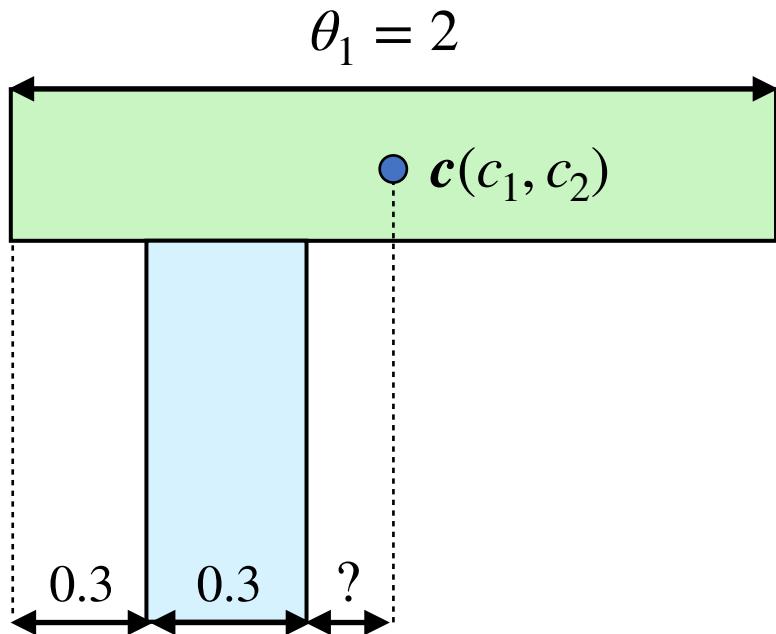


Infeasibility Energy

$$E = \| \text{---} \|^2$$
$$= (0.7 - \theta)^2$$

# Step #3 Sensitivity Analysis

- Compute the infeasibility energy's gradient/hessian with respect to the design parameters.



**Gradient:**  $\frac{\partial E}{\partial \theta_1}$

**Chain Rule:** 
$$\frac{\partial E}{\partial \theta_1} = \frac{\partial E}{\partial c_1} \frac{\partial c_1}{\partial \theta_1}$$

# Step #4 Numerical Optimization

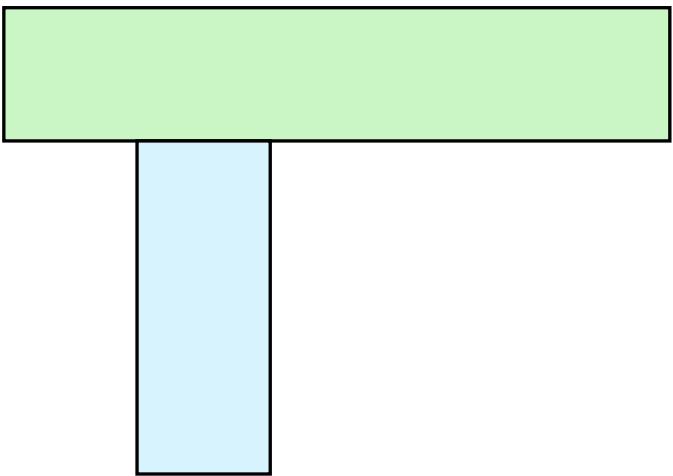
- Various numerical optimization tools can be used to solve the inverse design problem.

	Gradient Descent	Netwon Method	Quasi-Netwon Method
Data	Gradient	Hessian	Gradient
Speed	Slow	Fast	Medium
Code	Easy-to-implement	Hard-to-implement	Easy-to-implement

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## Part 2: Stability Optimization For Gravitational Equilibrium

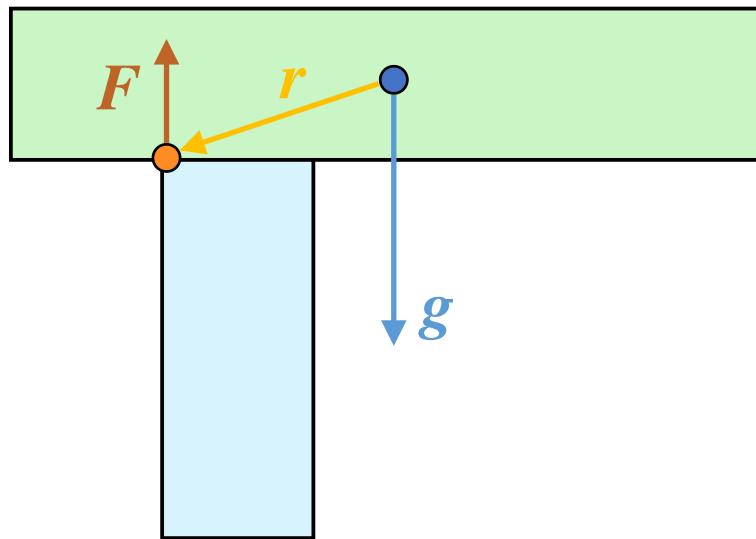
# Assumptions



1. Parts are rigid body.
2. Friction is ignored.
3. The bottom part (blue) is fixed.

# Recap: Rigid Body Equilibrium

- Rigid body equilibrium can check whether the internal and external forces/torque of a given structure are balanced.



Force Balance:  $\sum \textcolor{brown}{F} + \textcolor{blue}{g} = 0$

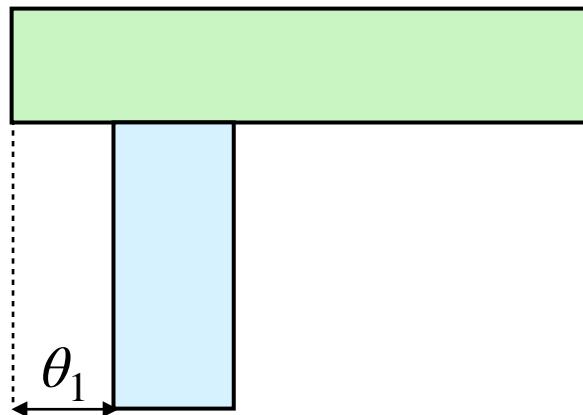
Torque Balance:  $\sum \textcolor{yellow}{r} \times \textcolor{brown}{F} = 0$

Non-negative:  $\textcolor{brown}{F} \geq 0$

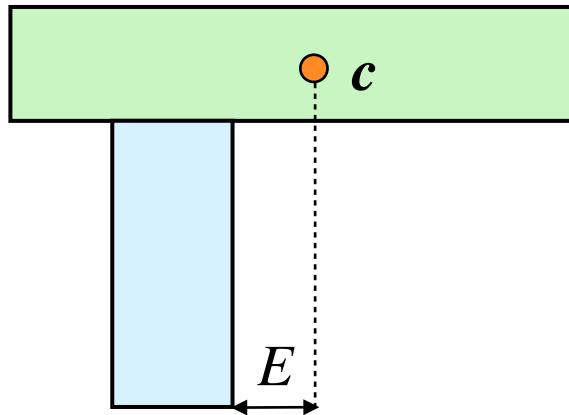
**Problems:** only provides a binary result (yes/no).

# Recap: Gradient-based Stability Optimization

- Compute faithful infeasibility energy.



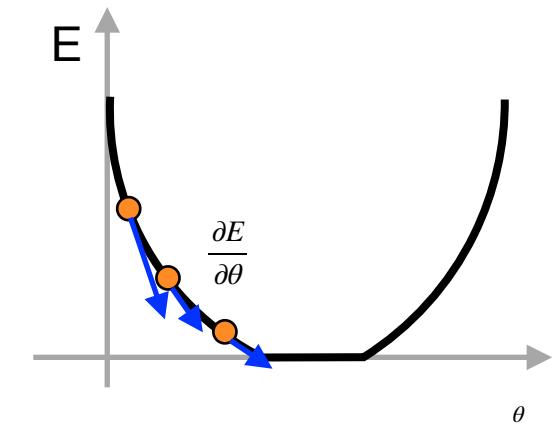
Step 1  
Geometrical Property



Step 2  
Infeasibility Energy

$$\theta \xrightarrow{\partial} c \xrightarrow{\partial} E$$

Step 3  
Sensitivity Analysis

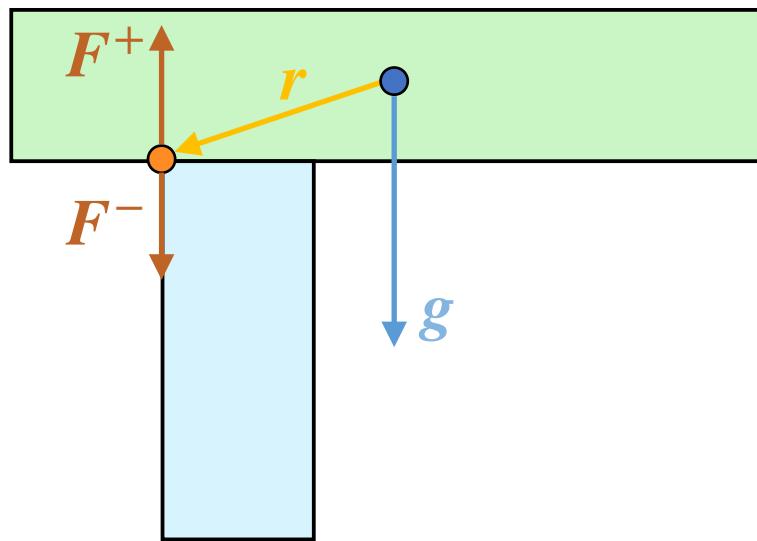


Step 4  
Numerical Optimization

# Equilibrium Infeasibility Energy

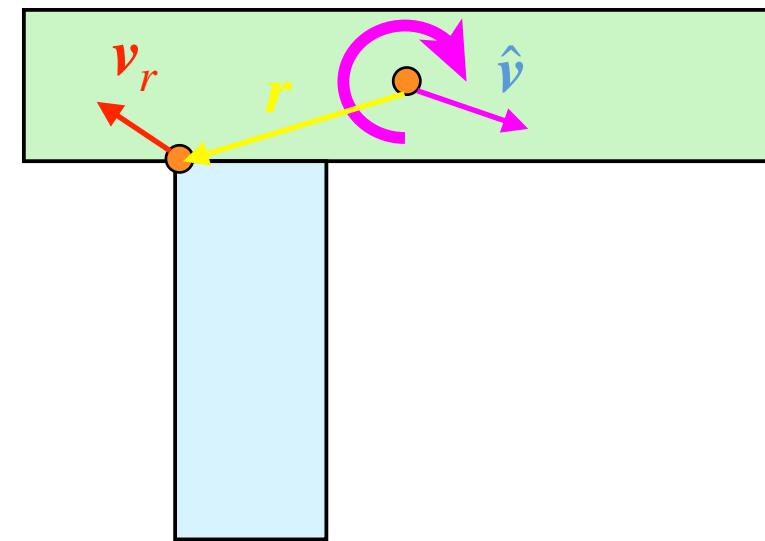
- Two ways of computing infeasibility energy for equilibrium problems.

**Force-based Equilibrium Method**



[Whiting et al 2009, 2012]

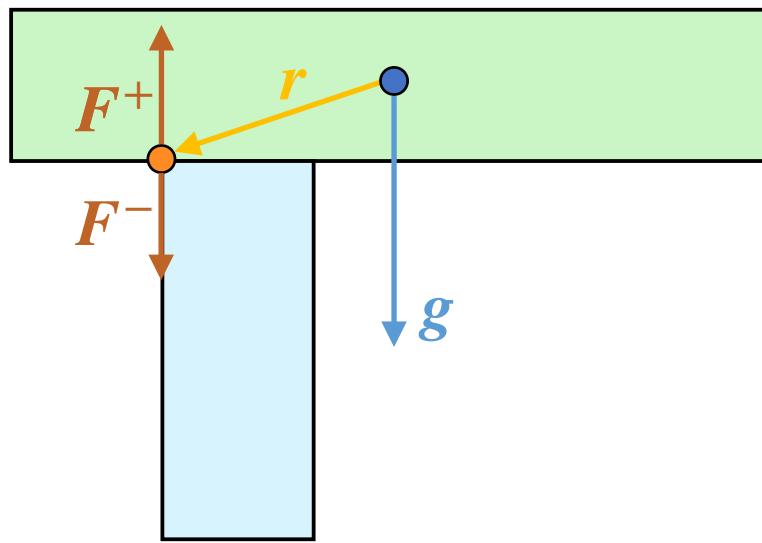
**Kinematic-based Equilibrium Method**



[Wang et al 2021]

# Force-based Infeasibility Measurement

- Split each contact force  $\mathbf{F}$  into the positive and negative parts  $\mathbf{F}^+$ ,  $\mathbf{F}^-$ .
- The norm of the negative contact force is used to compute the infeasibility energy.



[Whiting et al 2009, 2012]

Minimizing tension:  $\min \sum ||\mathbf{F}^-||^2$

Force Balance:  $\sum \mathbf{F} + \mathbf{g} = 0$

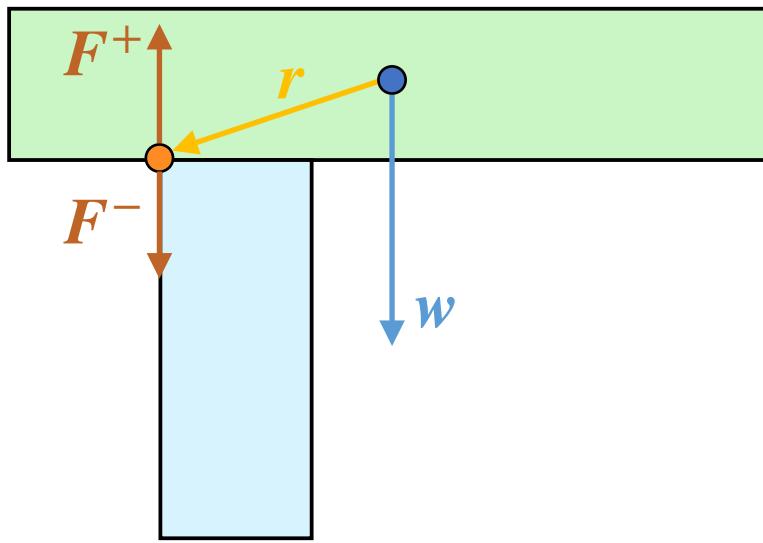
Torque Balance:  $\sum \mathbf{r} \times \mathbf{F} = 0$

Non-negative:  $\mathbf{F}^+, \mathbf{F}^- \geq 0$

$\mathbf{F} = \mathbf{F}^+ - \mathbf{F}^-$

# Quadratic Programming

- The infeasibility energy can be computed by a quadratic programming solver.



Minimizing tension:  $\min \sum ||F^-||^2$

Force/Torque Balance:  $A_{\text{eq}}F + w = 0$

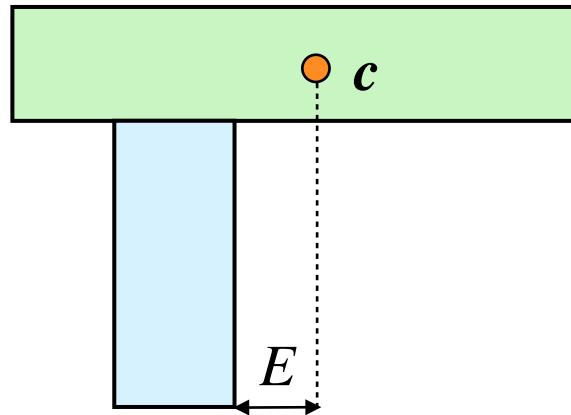
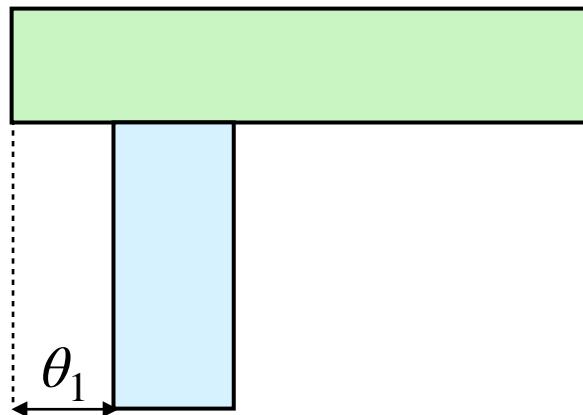
Non-negative:  $F^+, F^- \geq 0$

$$F = F^+ - F^-$$

[Whiting et al 2009, 2012]

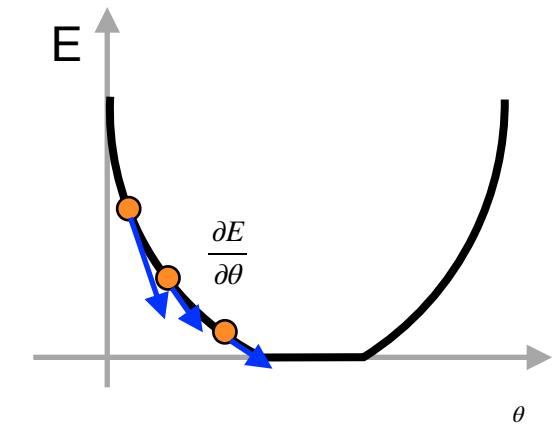
# Gradient-based Stability Optimization

- The next challenging step is to compute gradient using sensitivity analysis.



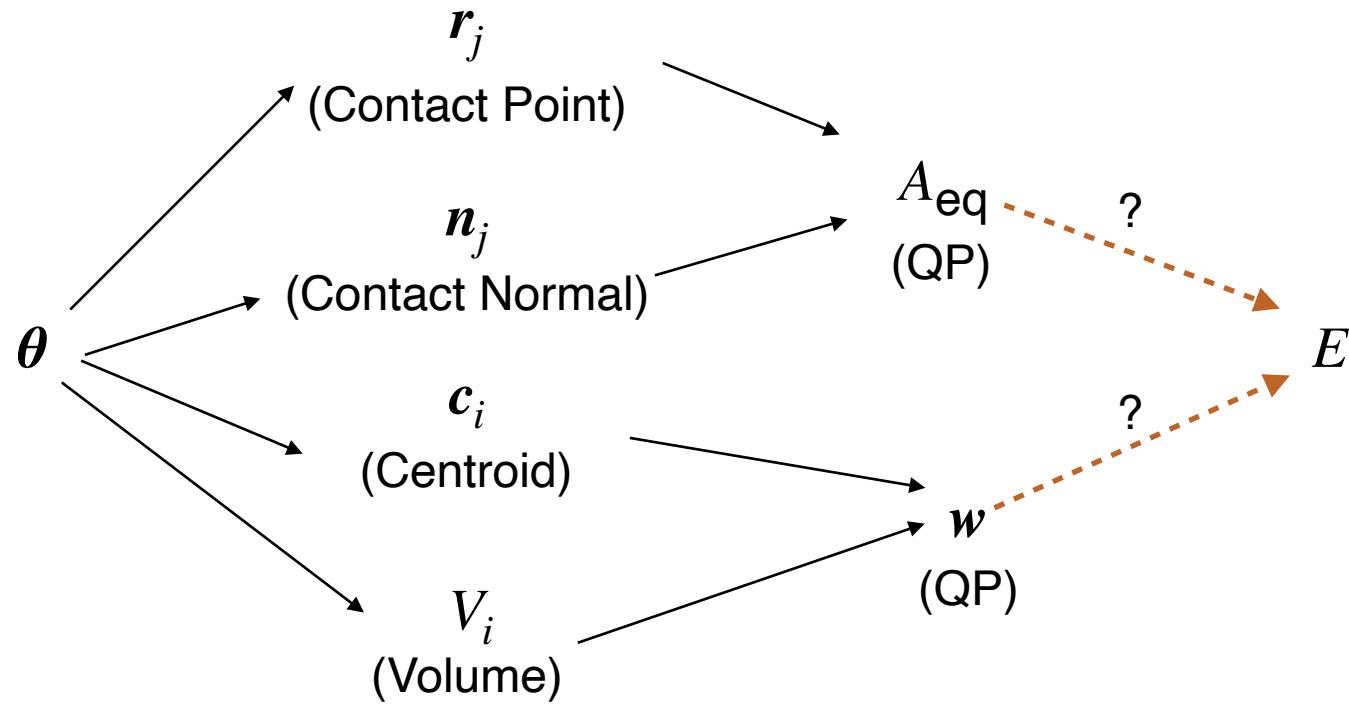
$$\theta \xrightarrow{\partial} c \xrightarrow{\partial} E$$

Step 3  
Sensitivity Analysis



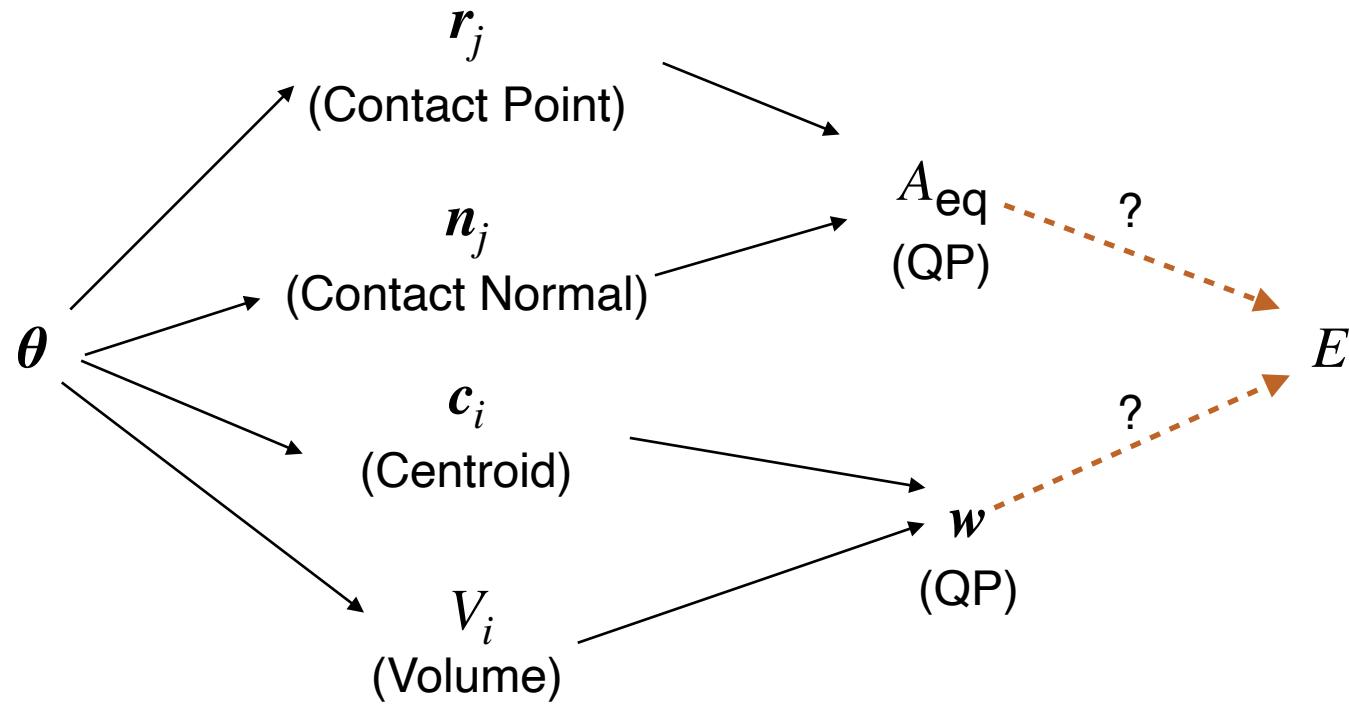
# Chain Rule

- The chain rule help compute the gradient.
- However, the infeasibility energy's gradient with respect to the QP's coefficients are missing.



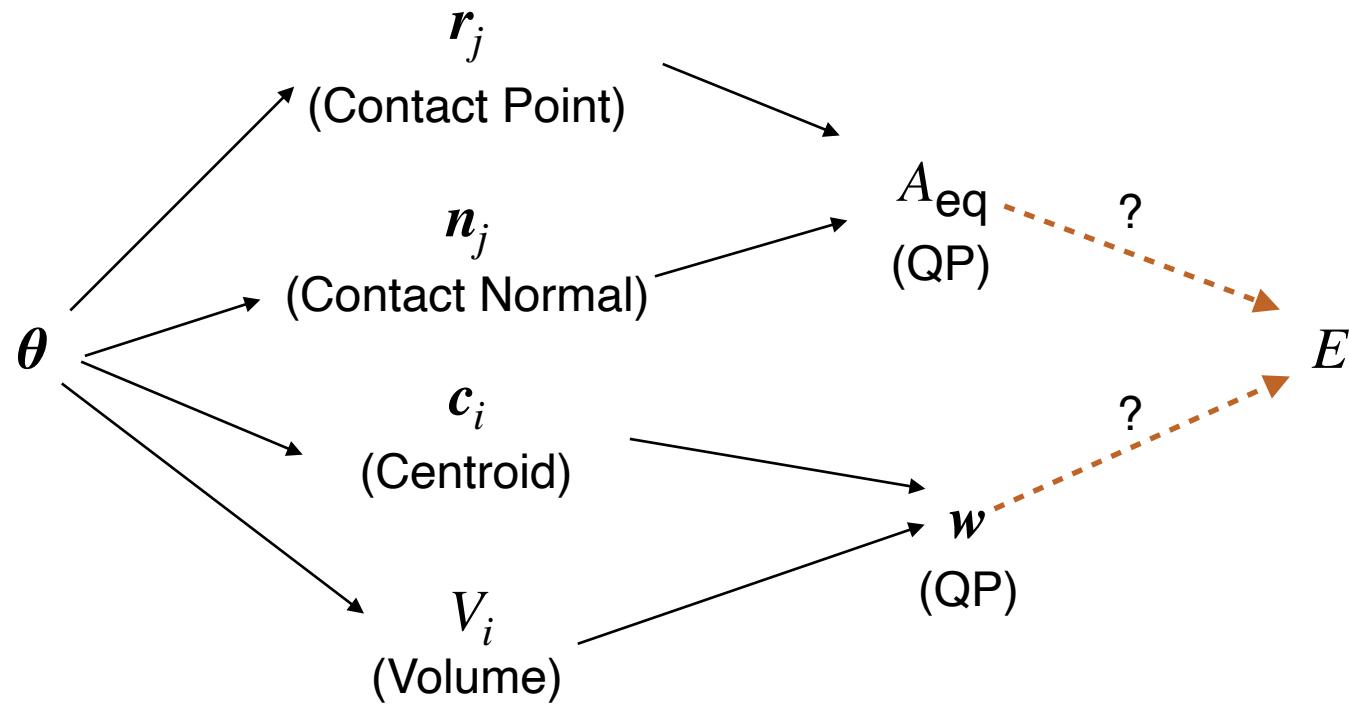
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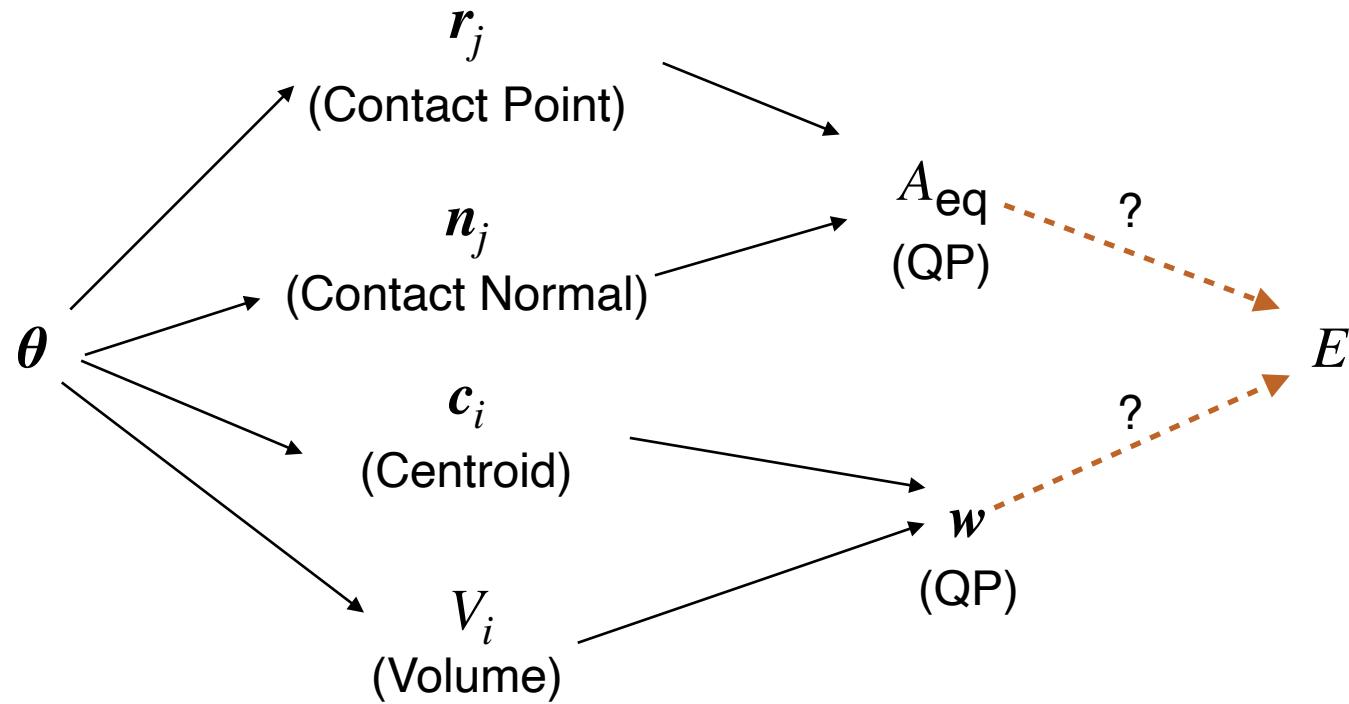
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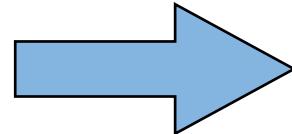


# Sensitivity Analysis of QP

- A closed-form solution is available for the QP problem with only equality constraints.

$$E(A_{\text{eq}}, w) = \min \sum ||F^-||^2$$

$$A_{\text{eq}} F + w = 0$$



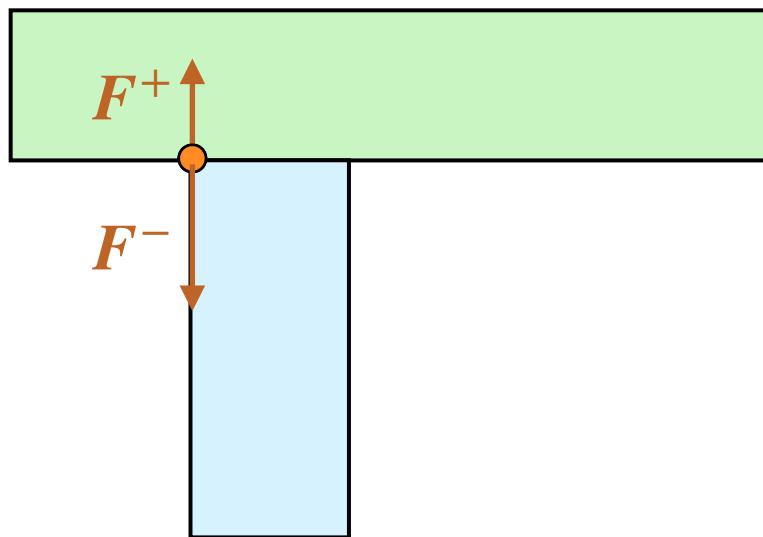
Closed-Form Solution

$$F^+, F^- \geq 0$$

[Whiting et al 2009, 2012]

# Sensitivity Analysis of QP

- Local perturbation of the geometry will only change the resulting force slightly.



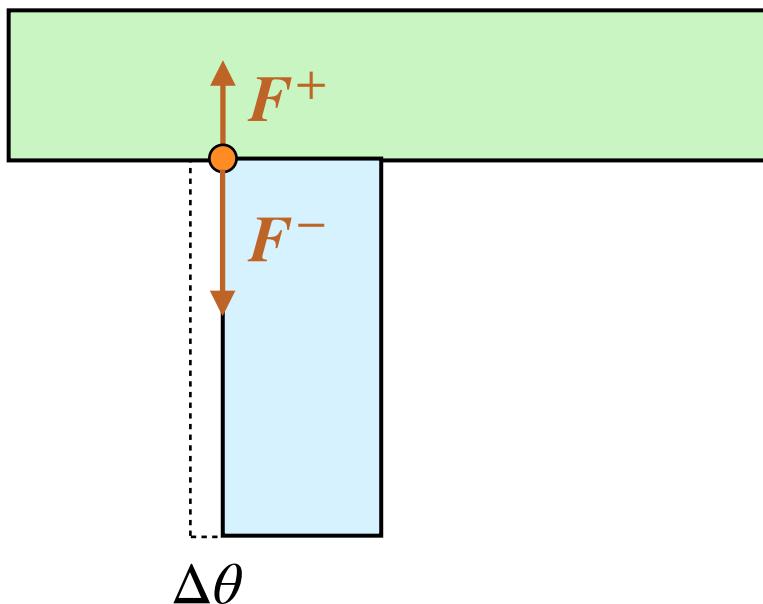
$$F^+ = 1.0$$

$$F^- = 1.5$$

[Whiting et al 2009, 2012]

# Sensitivity Analysis of QP

- Local perturbation of the geometry will only change the resulting force slightly.



$$F^+ = 1.01$$

$$F^- = 1.49$$

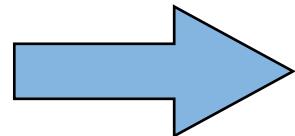
[Whiting et al 2009, 2012]

# Sensitivity Analysis of QP

- Applying region trust algorithm to replace inequalities with equalities.

$$E(A_{\text{eq}}, w) = \min \sum ||F^-||^2$$

$$A_{\text{eq}} F + w = 0$$



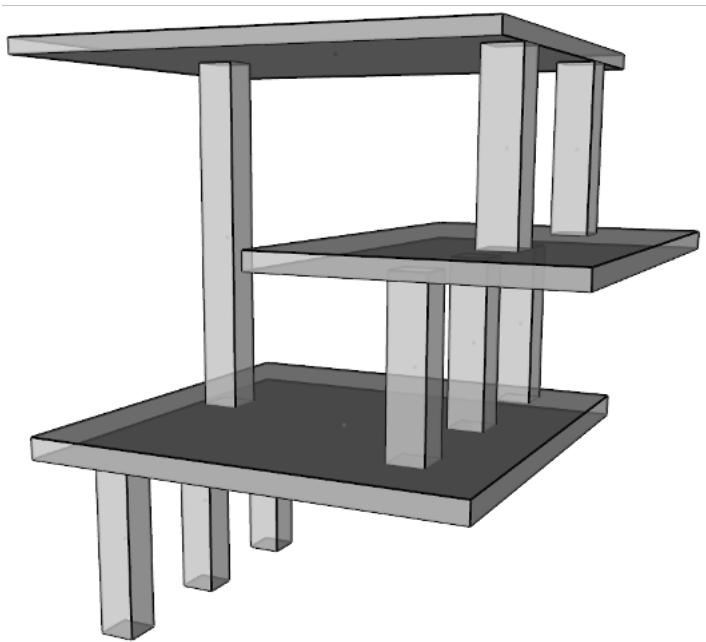
Closed-Form Solution

$$F^+, F^- \geq 0$$

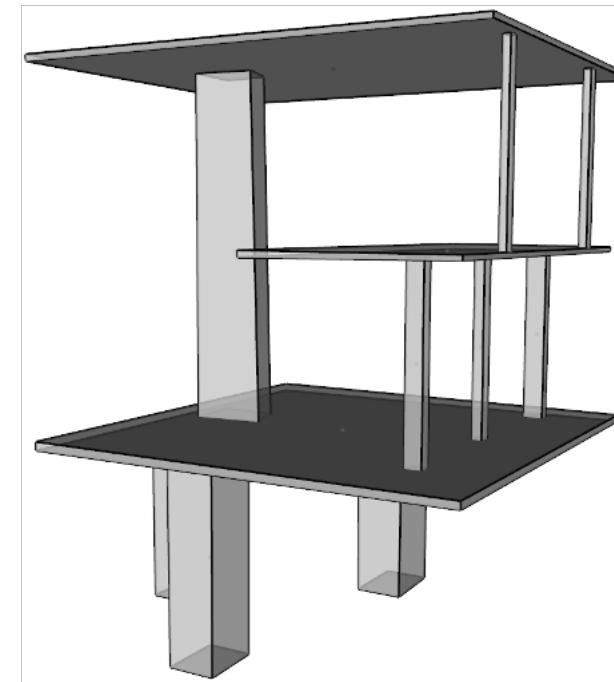
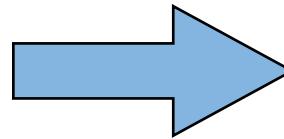
$$F_i^+ = 0, \quad F_j^- = 0$$

[Whiting et al 2009, 2012]

# Result



Unstable output

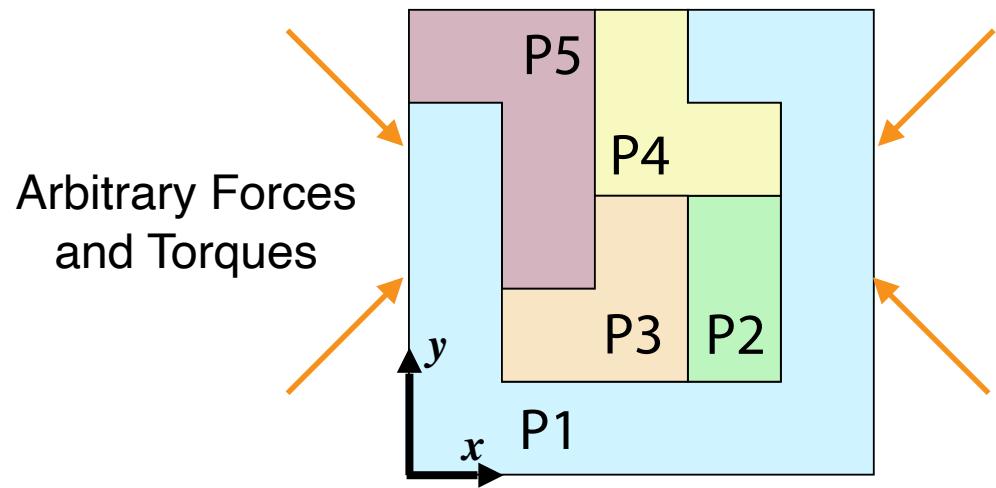


Stable output

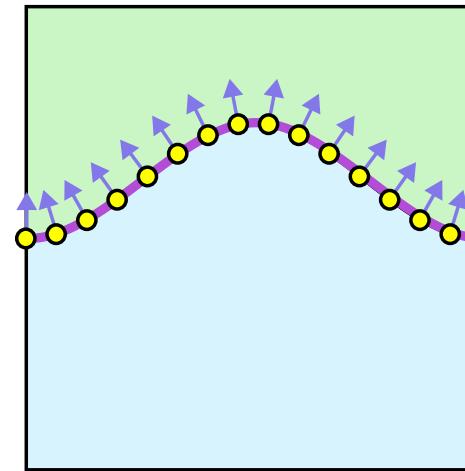
[Whiting et al 2012]

# Limitations of Force-based Method

- Hard to test for some stability types (i.e., globally interlocking)
- Less efficient when handling parts with non-planar contacts.



Globally Interlocking Assemblies

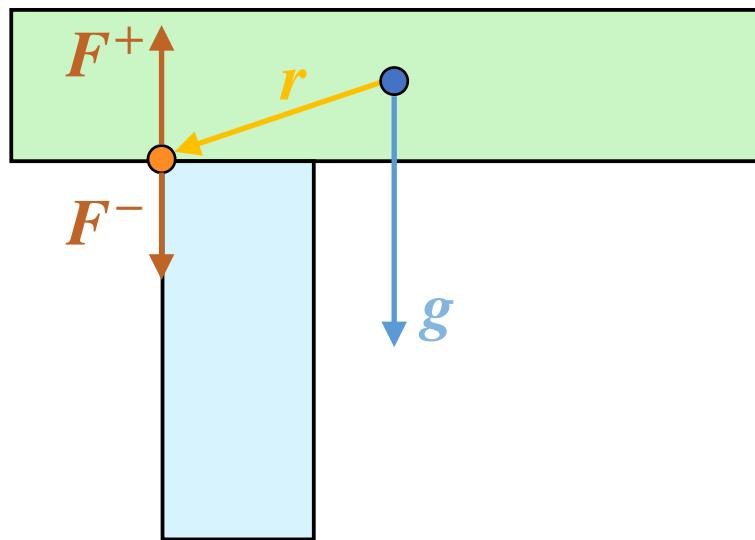


Curve Contacts

# Kinematic-based Equilibrium Method

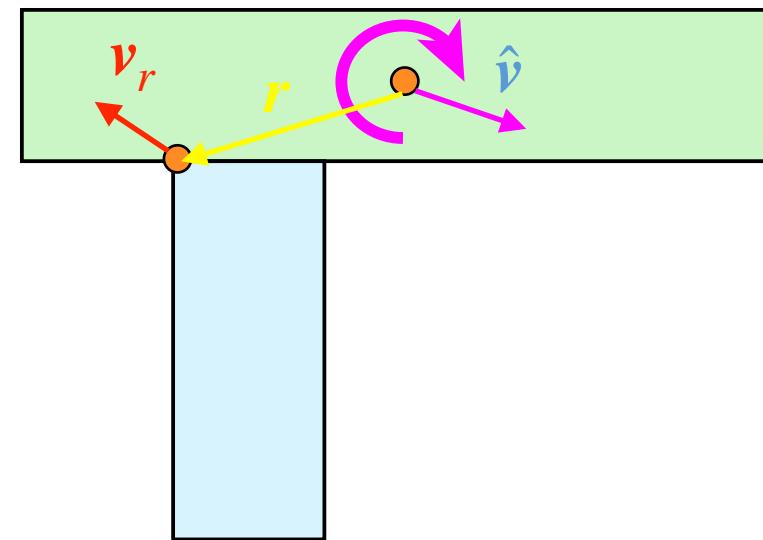
- Kinematic-based method measures infeasibility in the motion space.

**Force-based Equilibrium Method**



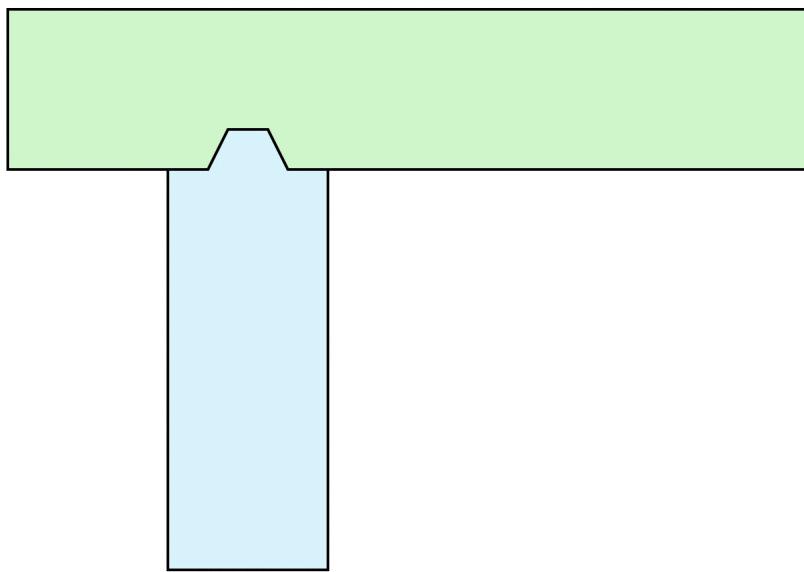
[Whiting et al 2009, 2012]

**Kinematic-based Equilibrium Method**



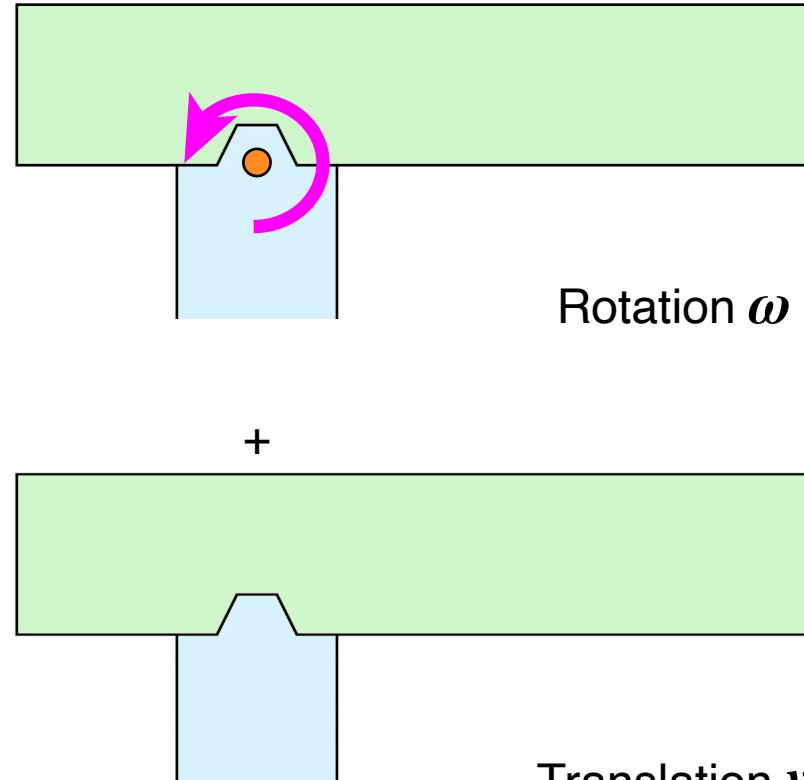
[Wang et al 2021]

# Infinitesimal Rigid Motion

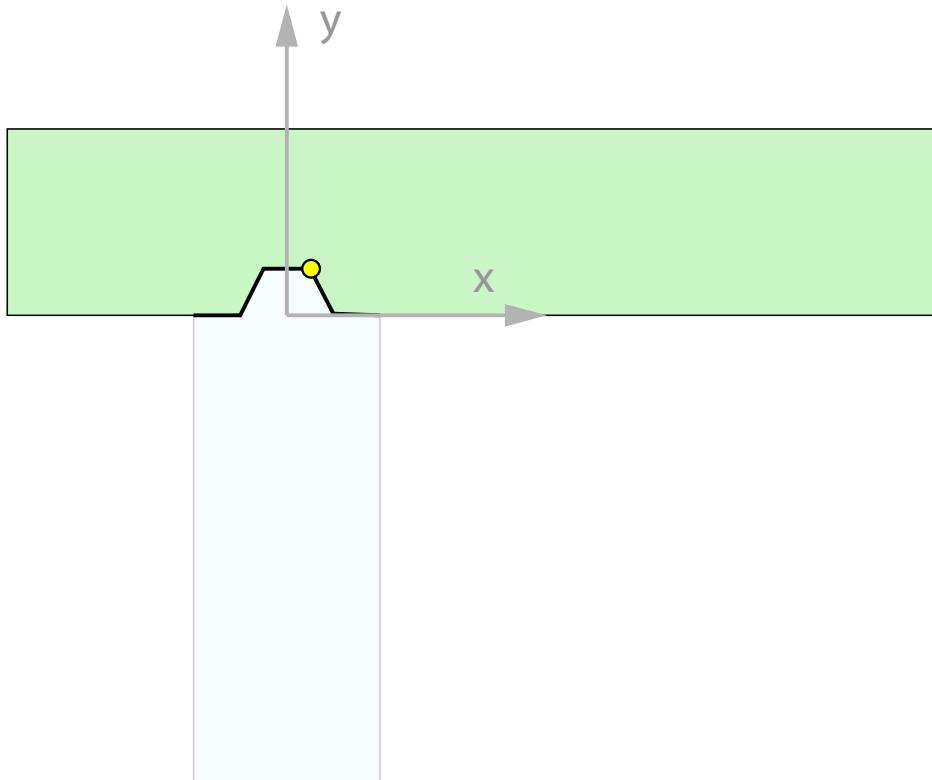


Infinitesimal rigid motion  $\hat{v} = (v, \omega)$

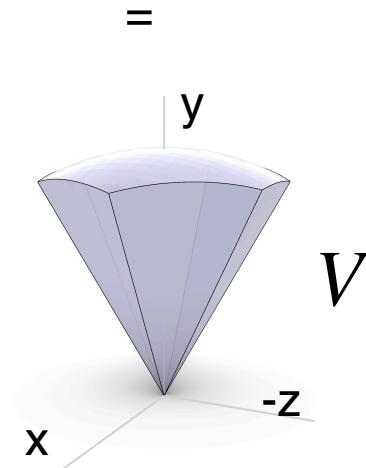
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# Motion Space

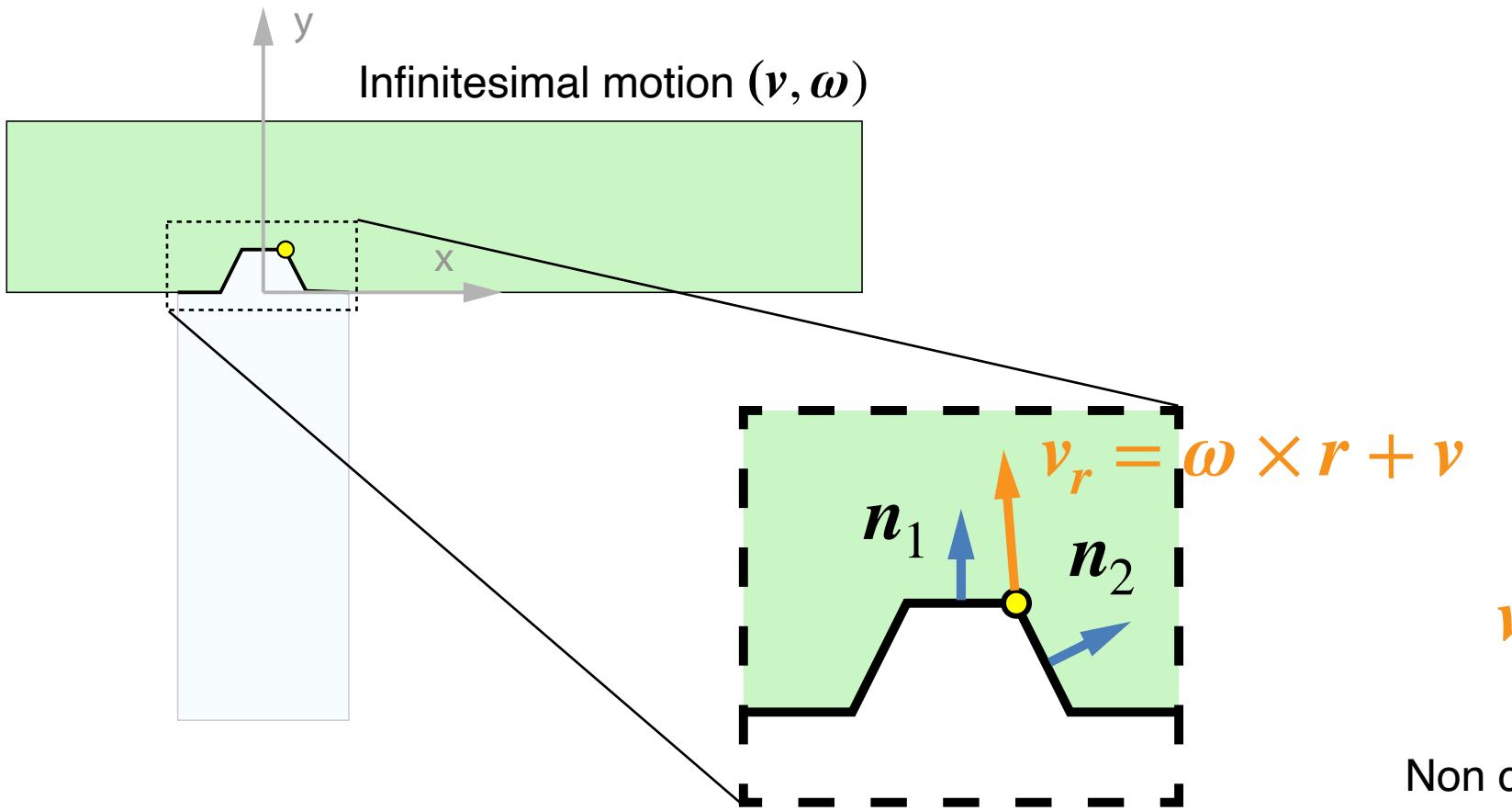


The motion space  $V$  of green part



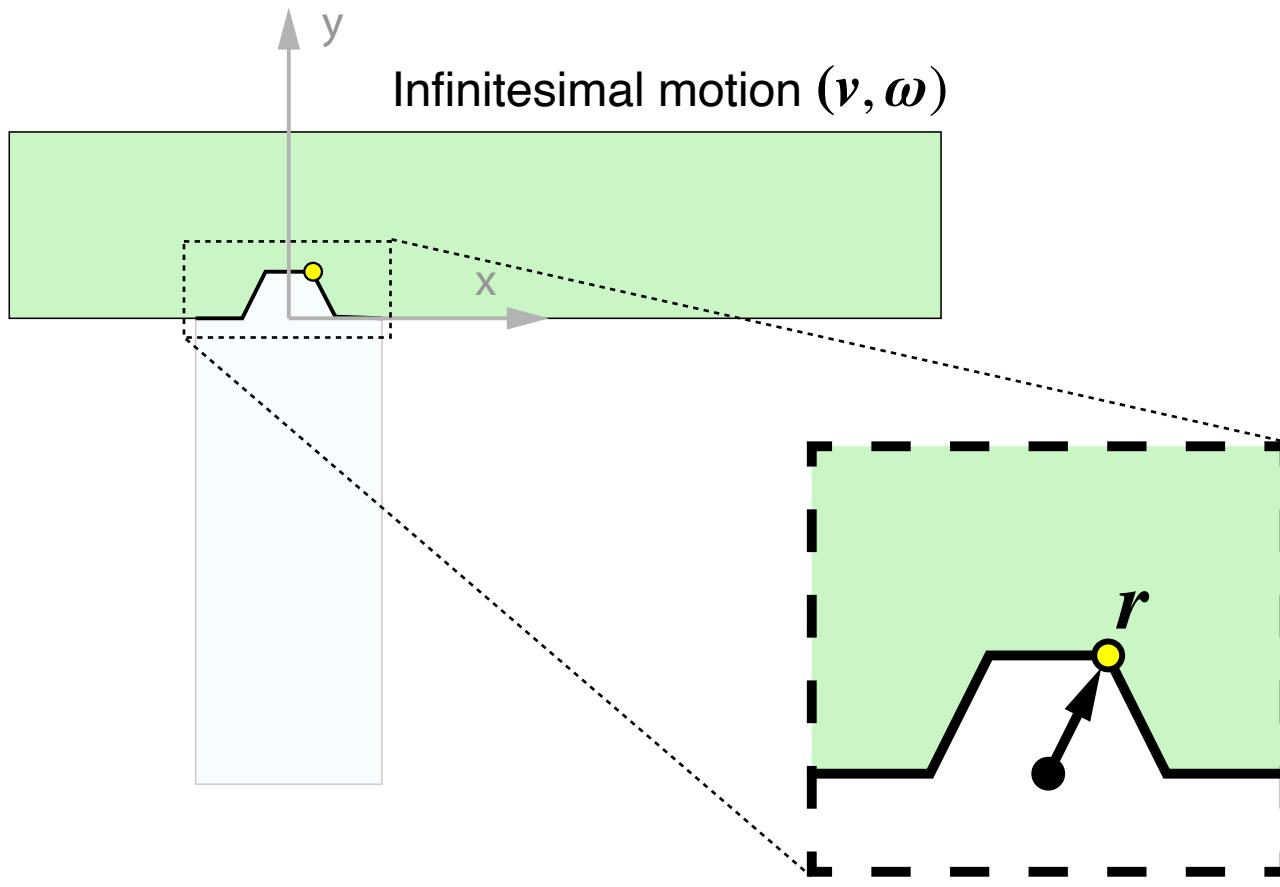
{collision-free infinitesimal rigid motions  $\hat{v}$ }

# Non-collision constraints



$$v_r \cdot n \geq 0$$

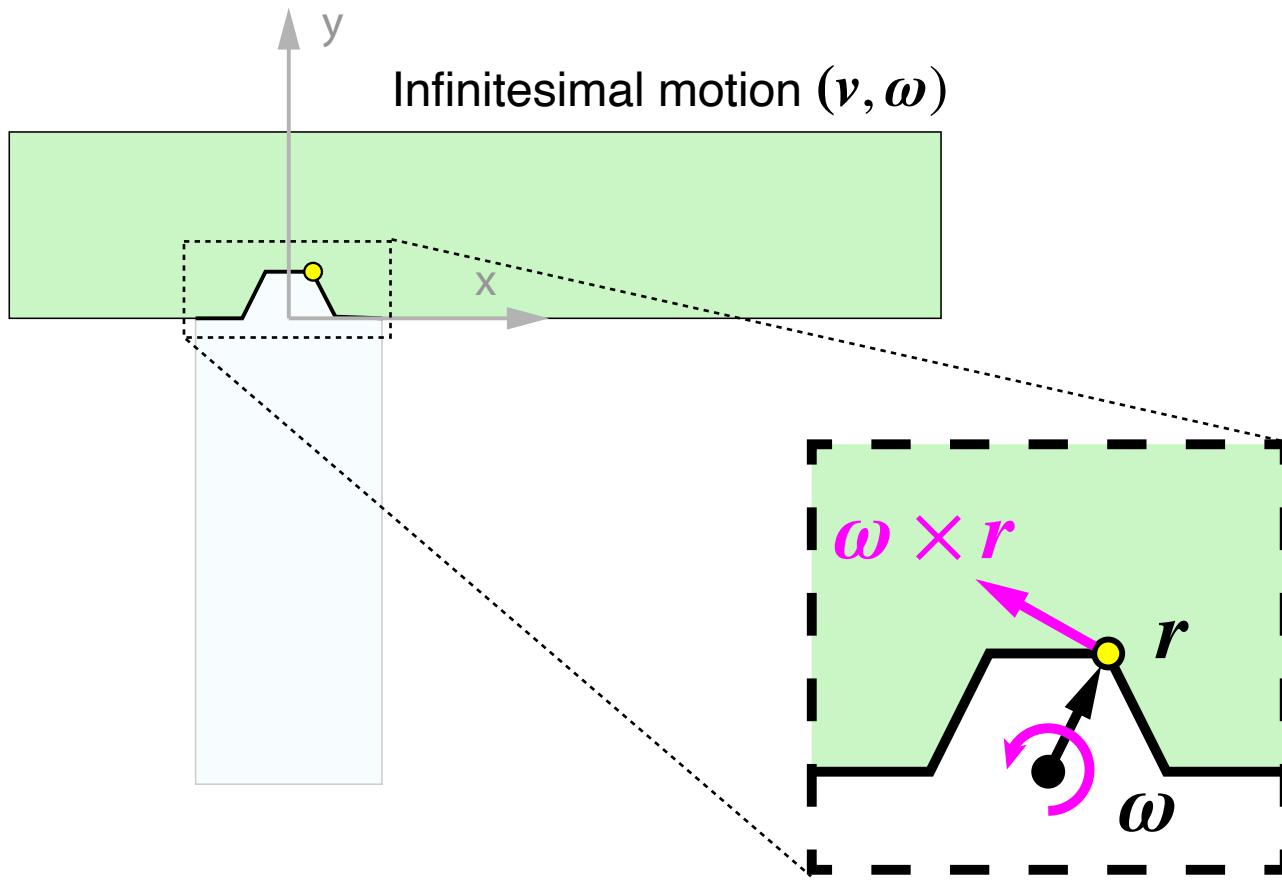
# Contact Points' Velocities



velocity  $v_r$

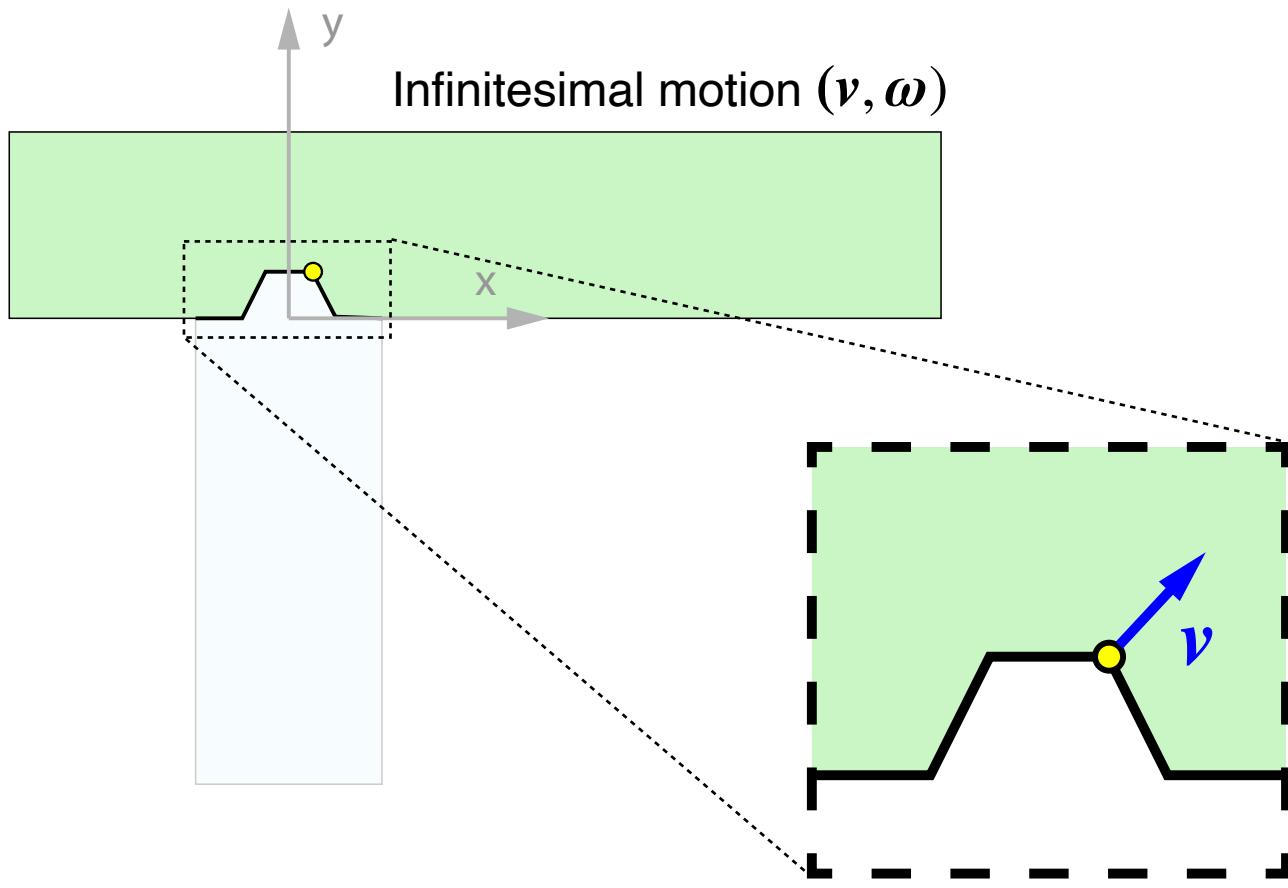
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# Contact Points' Velocities



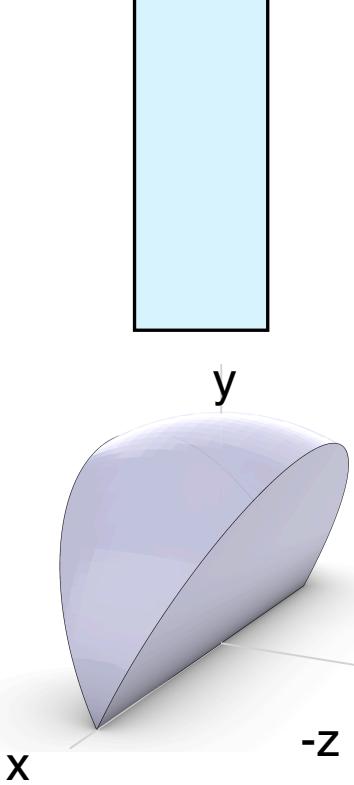
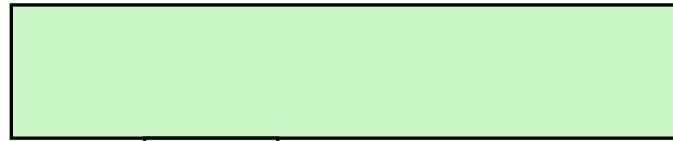
$$\begin{aligned} \text{velocity } v_r \\ = \\ \text{rotation } \omega \times r \\ + \end{aligned}$$

# Contact Points' Velocities



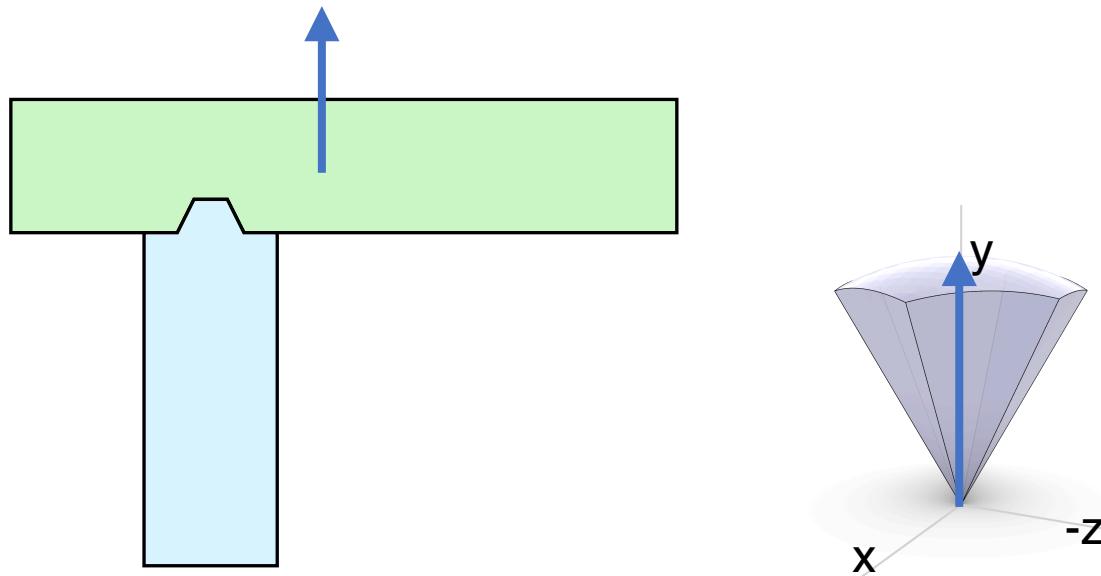
$$\begin{aligned} \text{velocity } v_r \\ = \\ \text{rotation } \omega \times r \\ + \\ \text{Translation } v \end{aligned}$$

# Motion Cone of Contacts



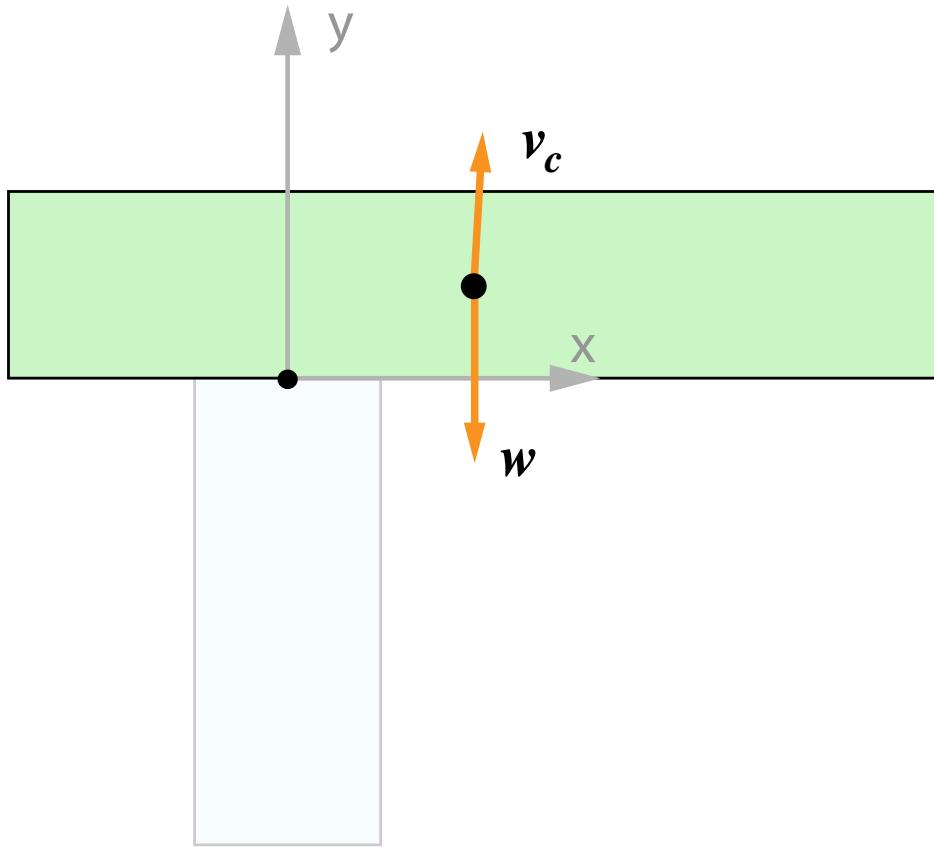
# Physically Feasible Motion

- Not every motion in the motion cone is physically plausible.



The translation along +y direction is not physically achievable.

# Feasible Motion Space



Velocity  $v_c$  at part's centre of mass  
decreases its gravitational potential

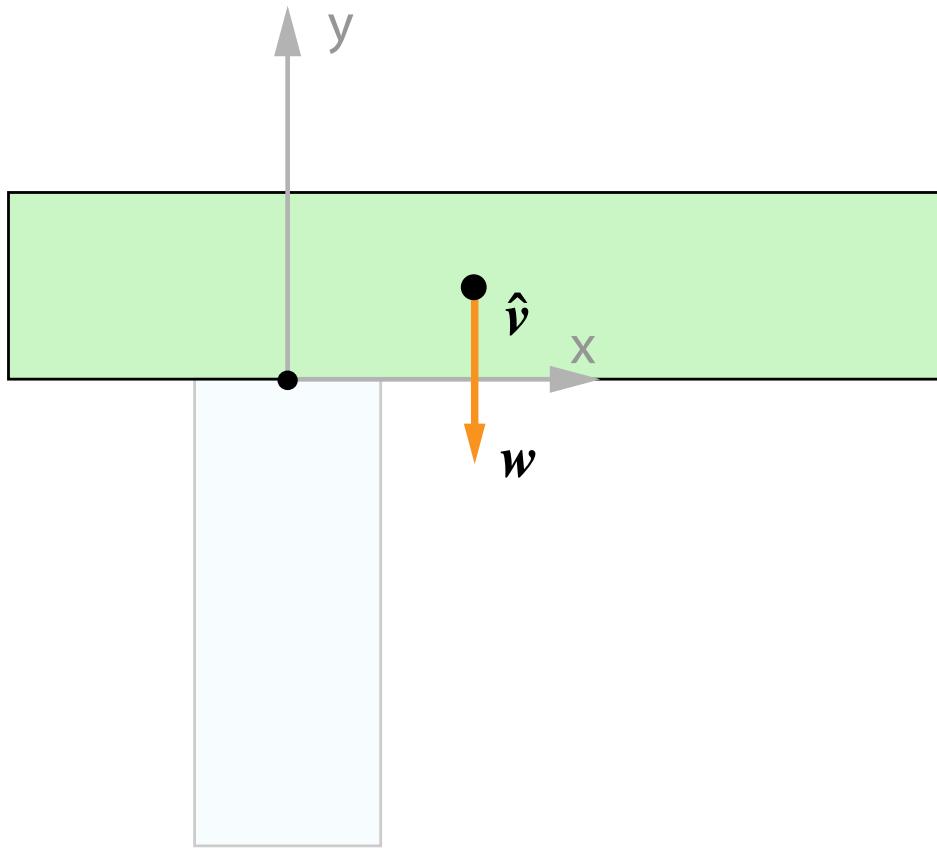
$$v_c \cdot g > 0$$



$$\hat{v} \cdot w > 0$$

Infinitesimal motion  $\hat{v} = (v, \omega)$

# Feasible Motions



Assembly is in equilibrium when

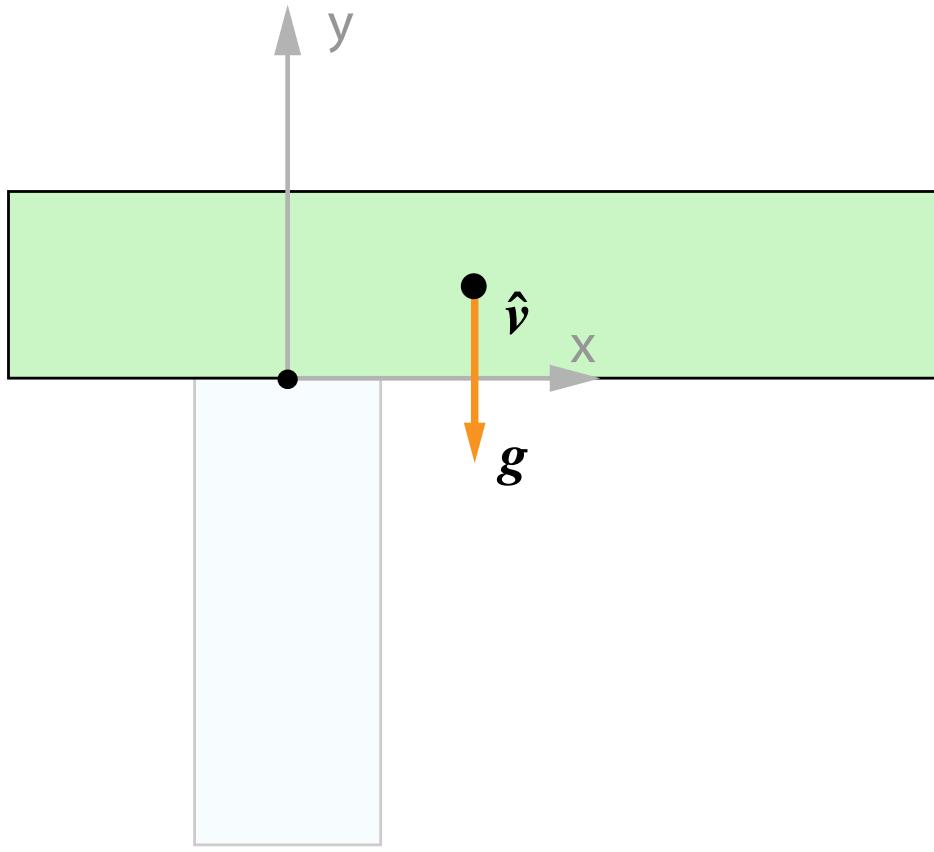
$$\left\{ \begin{array}{l} \hat{v} \cdot w > 0 \\ \hat{v} \in \text{Motion Cone} \end{array} \right.$$

A diagram of a motion cone, which is a three-dimensional shape resembling a double-cone or a V-shape opening upwards, centered at the origin of the coordinate system.

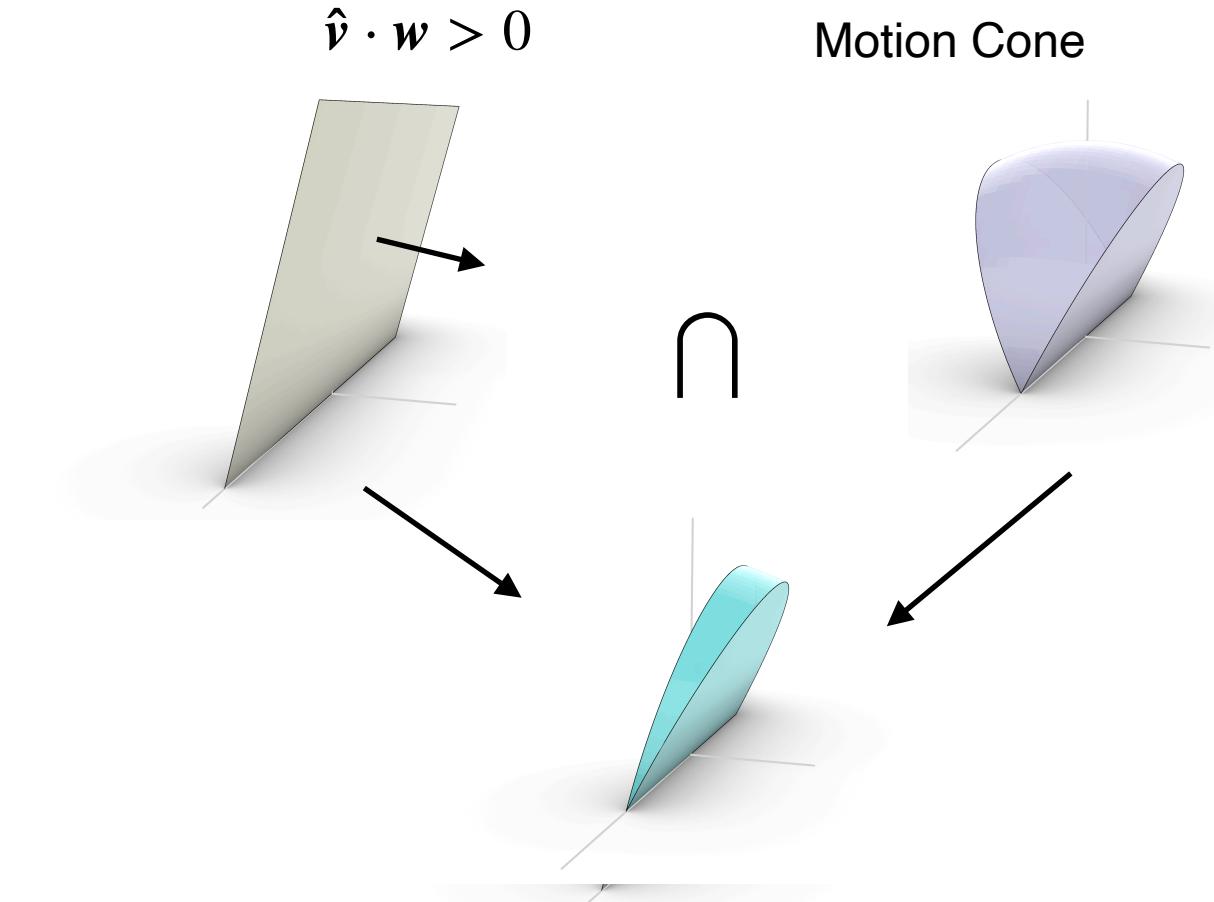
does not have solutions.

Infinitesimal motion  $\hat{v} = (v, \omega)$

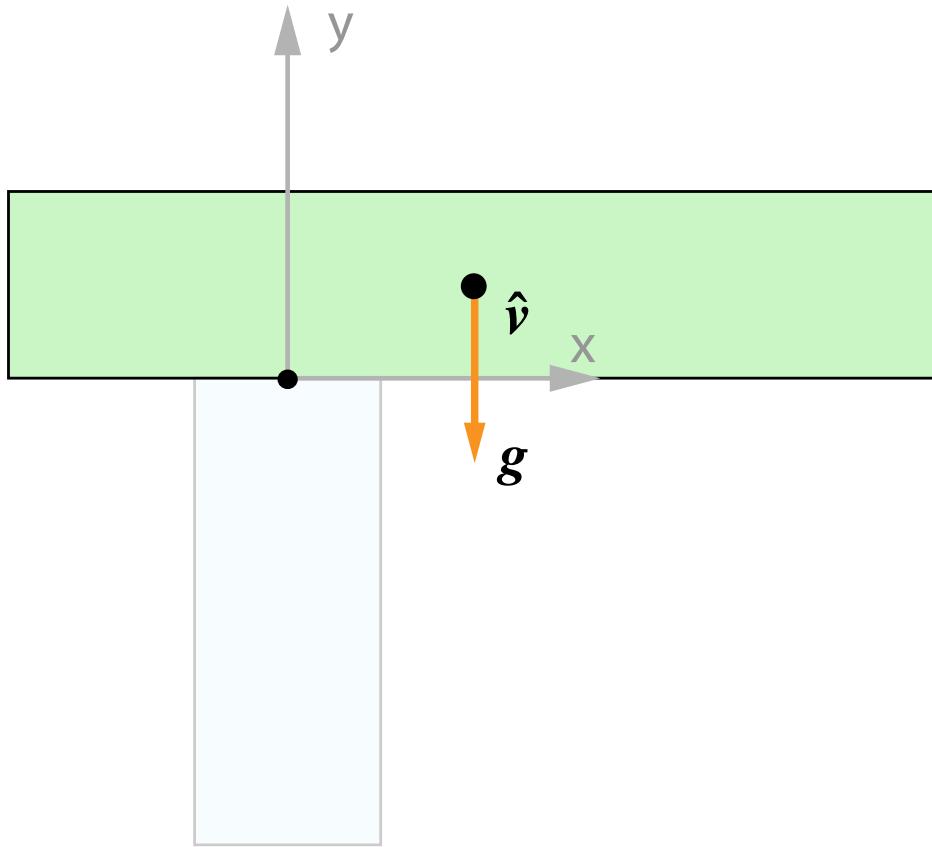
# Feasible Motion Space



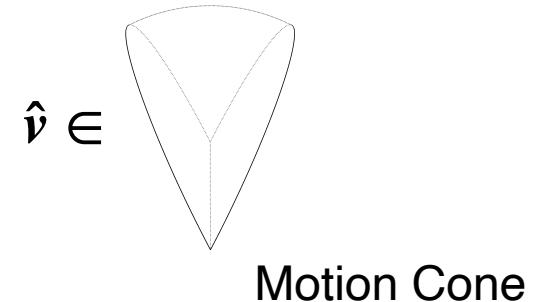
Infinitesimal motion  $\hat{v} = (v, \omega)$



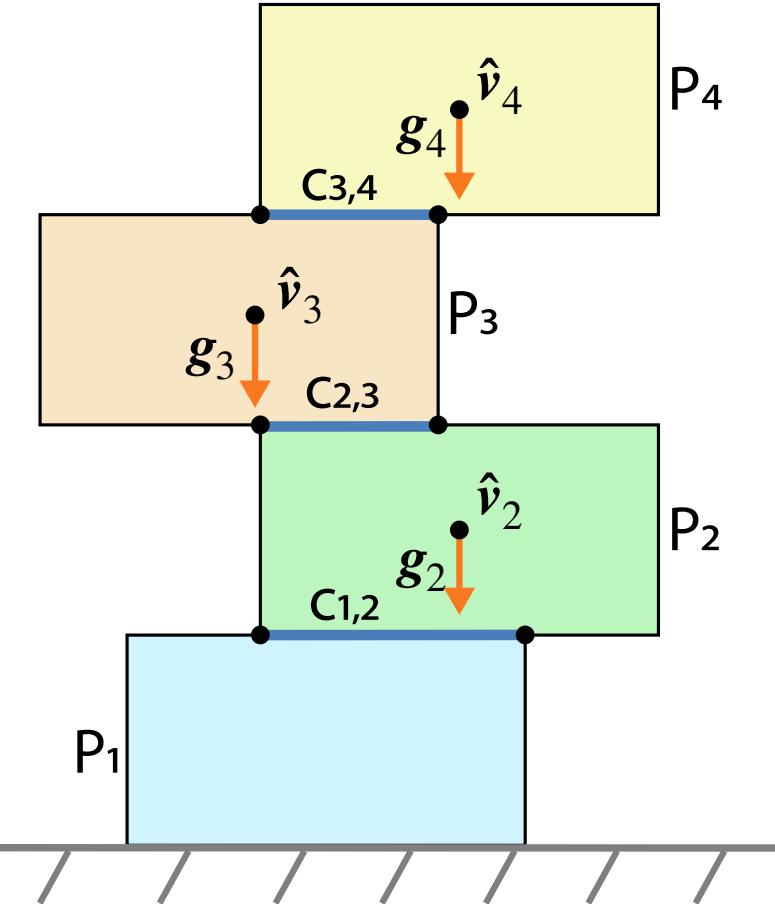
# Infeasibility Measurement



$$\max_{\hat{v}} \quad w \cdot \hat{v} - \frac{1}{2} \hat{v} \cdot \hat{v}$$



# Infeasibility Measurement for Assembly



$$\hat{v} = \begin{bmatrix} \hat{v}_2 \\ \hat{v}_3 \\ \hat{v}_4 \end{bmatrix}$$

$$\hat{g} = \begin{bmatrix} \hat{g}_2 \\ \hat{g}_3 \\ \hat{g}_4 \end{bmatrix}$$

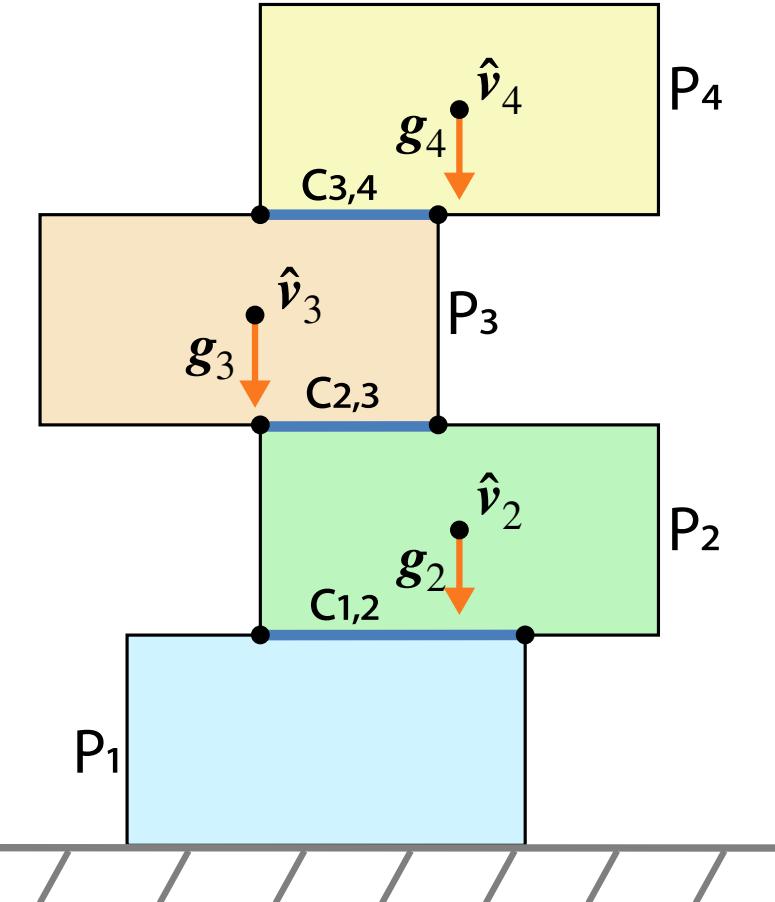
$$\max w \cdot \hat{v} - \frac{1}{2} \hat{v} \cdot \hat{v}$$

$$\hat{v}_2 \in V(C_{1,2})$$

$$\hat{v}_3 - \hat{v}_2 \in V(C_{2,3})$$

$$\hat{v}_4 - \hat{v}_3 \in V(C_{3,4})$$

# Infeasibility Measurement for Assembly



$$\hat{v} = \begin{bmatrix} \hat{v}_2 \\ \hat{v}_3 \\ \hat{v}_4 \end{bmatrix}$$

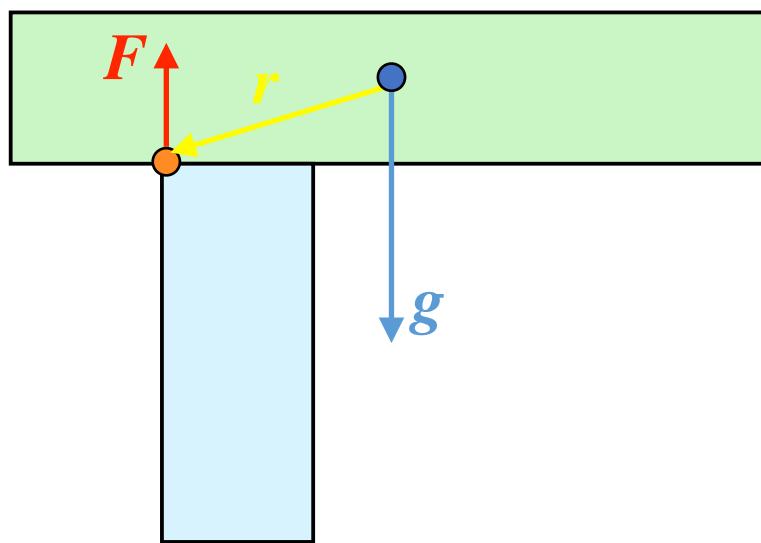
$$\hat{g} = \begin{bmatrix} \hat{g}_2 \\ \hat{g}_3 \\ \hat{g}_4 \end{bmatrix}$$

$$\max w \cdot \hat{v} - \frac{1}{2} \hat{v} \cdot \hat{v}$$

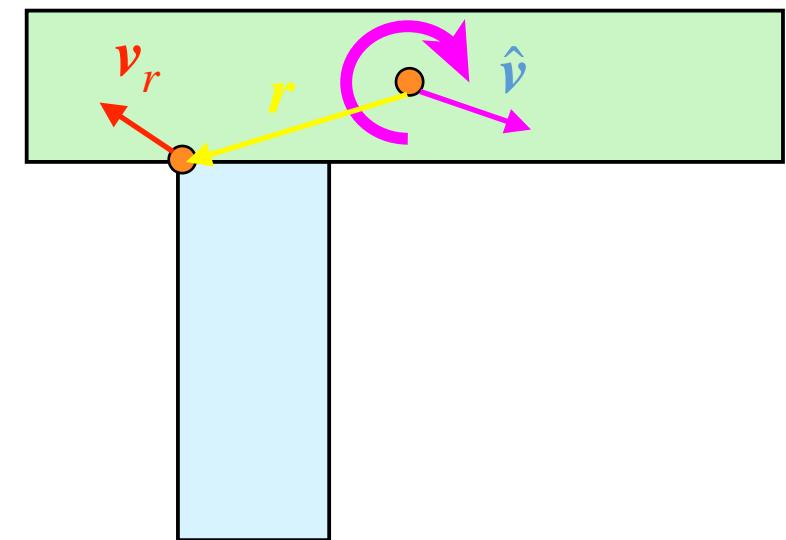
$$\text{s.t. } B_{\text{in}} \hat{v} \geq 0$$

# Static-Kinematic Duality

- The correctness of the kinematic-based method is due to the static-kinematic duality.



Statics

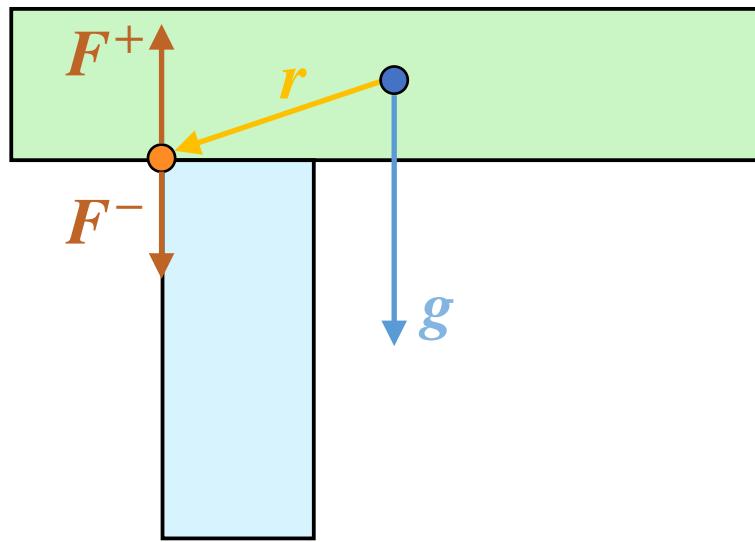


Kinematics

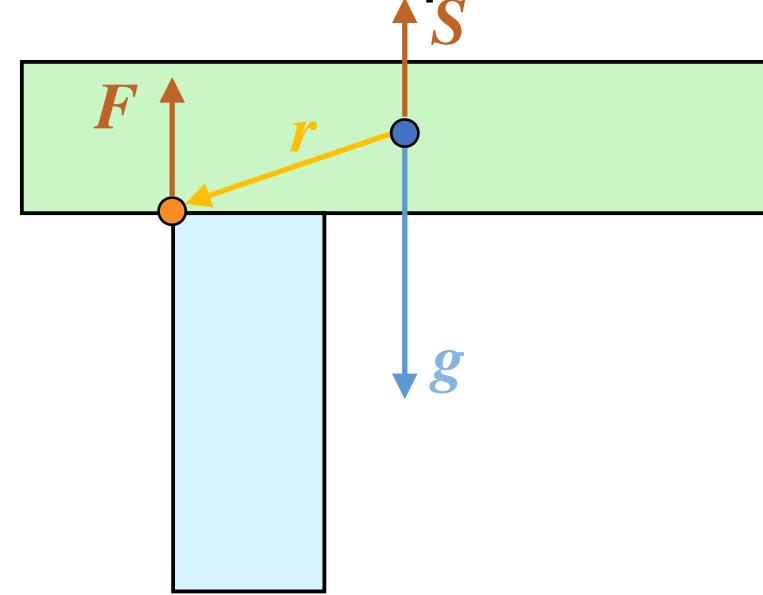
# Static-Kinematic Duality

- The kinematic-based method can be reformulated using forces.

Force-based Equilibrium Method



Kinematic-based Equilibrium Method



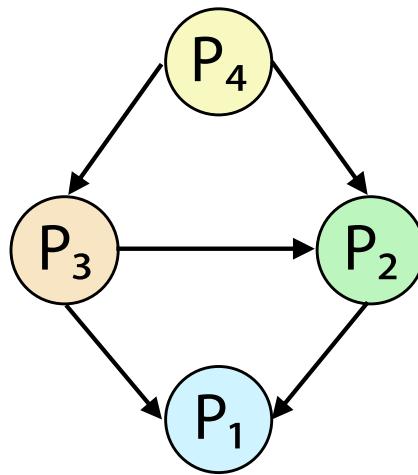
Reformulate:

Non-negative Condition

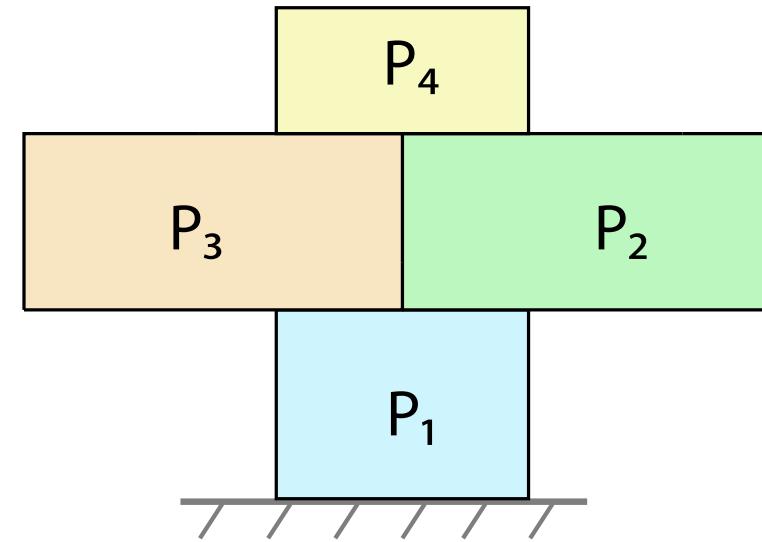
Force/Torque Balance Condition

# Representation for Stability Analysis

- Both representations have their own drawbacks.



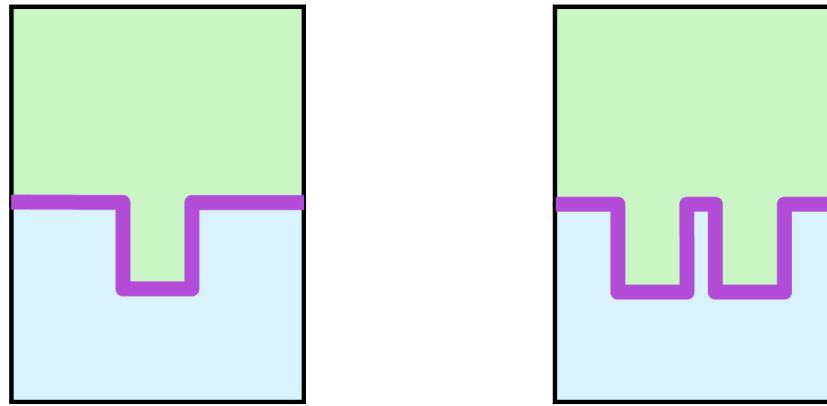
Part Graph



Part Geometry

# Geometric-based Representation

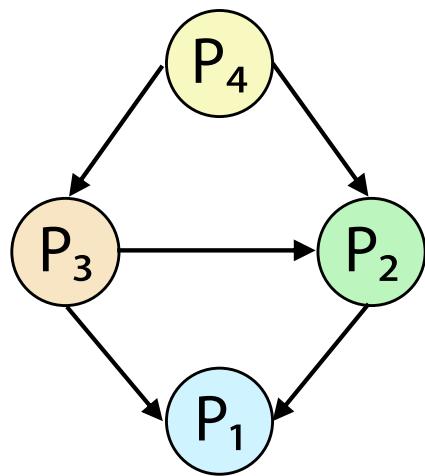
- Geometric-based Representation may have redundancy.



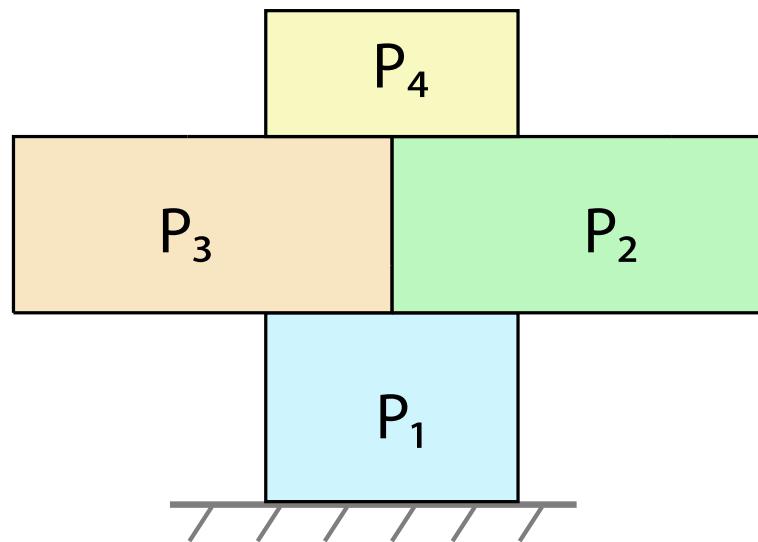
The two assemblies have the same structural stability.

# Graph-based Representation

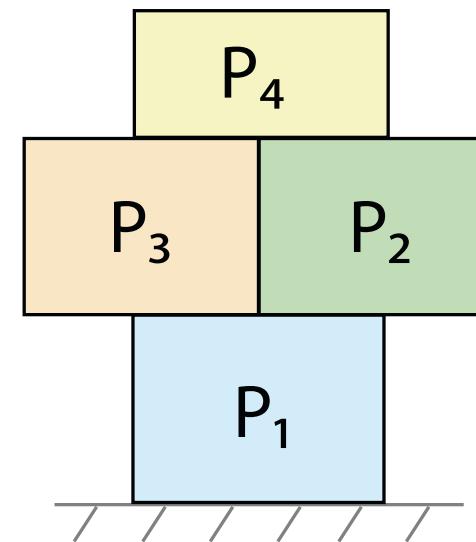
- Graph-based representation is not adequate for structural stability analysis.



Part Graph



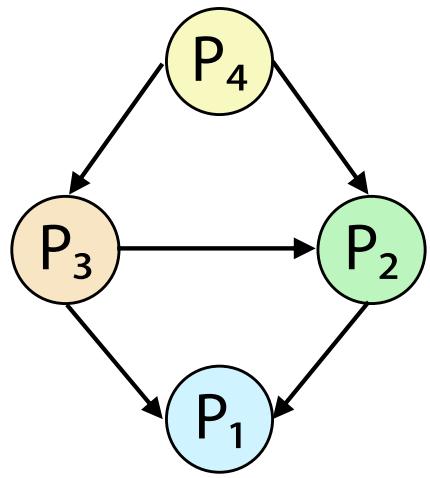
Unstable Assembly



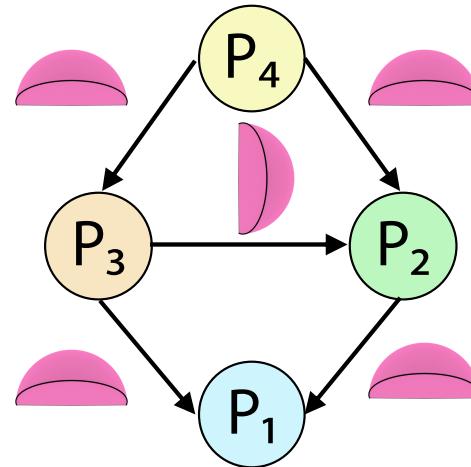
Stable Assembly

# Motion-based Representation

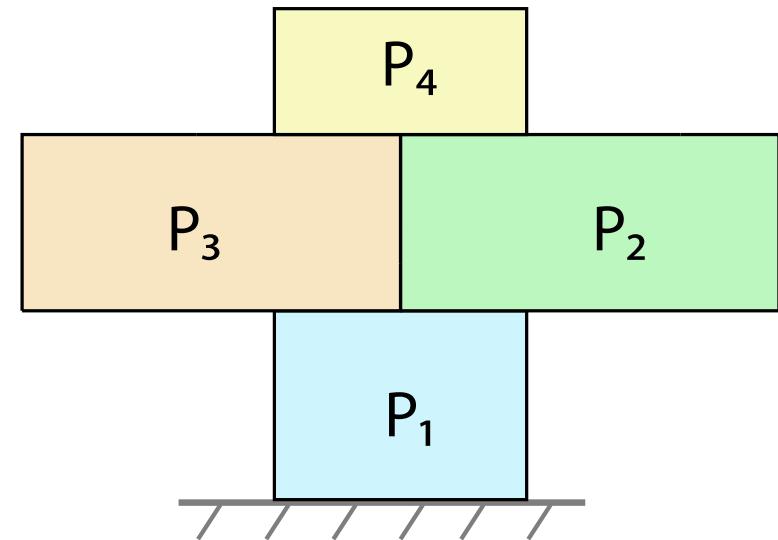
- We propose a motion-based representation which is a condensed representation for measuring structural stability of assemblies.



Part Graph



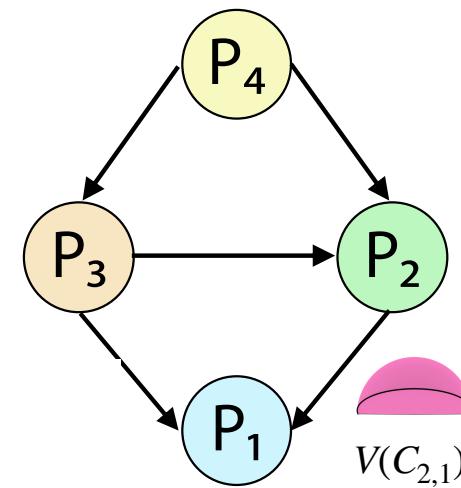
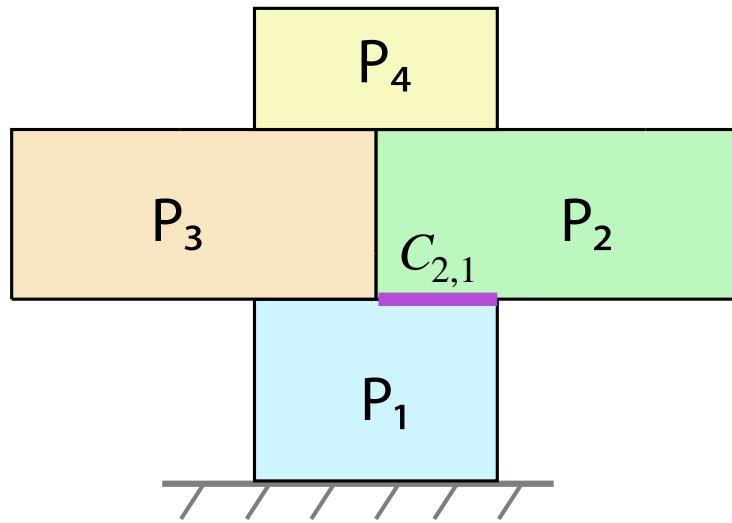
Graph-based Representation



Part Geometry

# Motion-based Representation

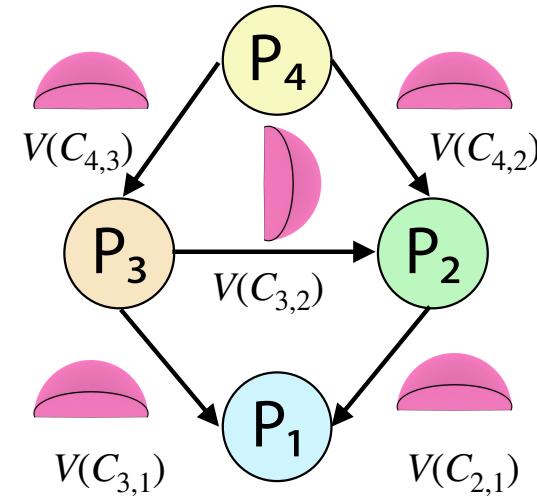
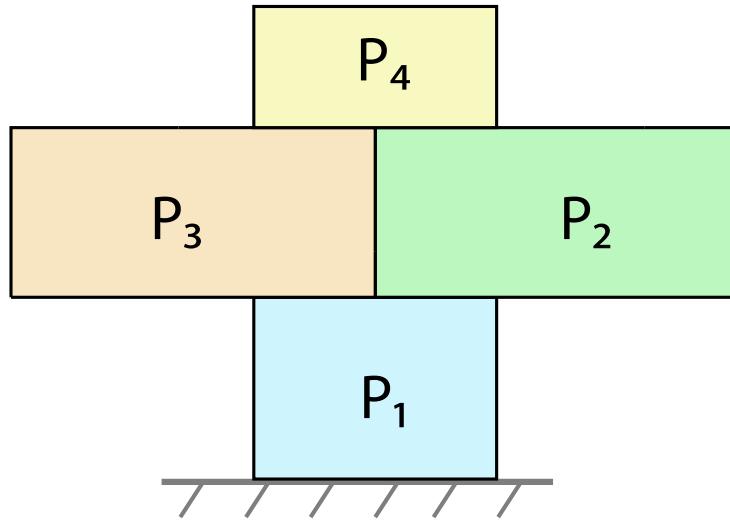
- Our motion-based representation is an augmented part graph with motion cones at its edges.



\*Arrow means  $P_2$  is installed after  $P_1$

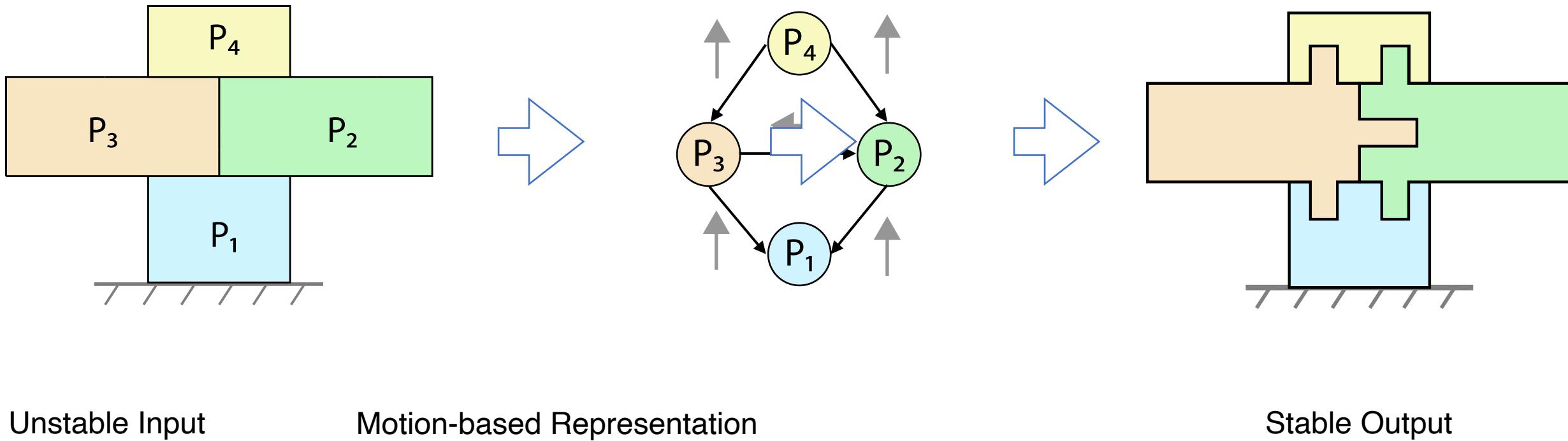
# Motion-based Representation

- Because of the duality between statics and kinematics, our motion-based representation can test for equilibrium.



# Kinematic-Geometric Design Framework

- Decoupling motion and geometry.

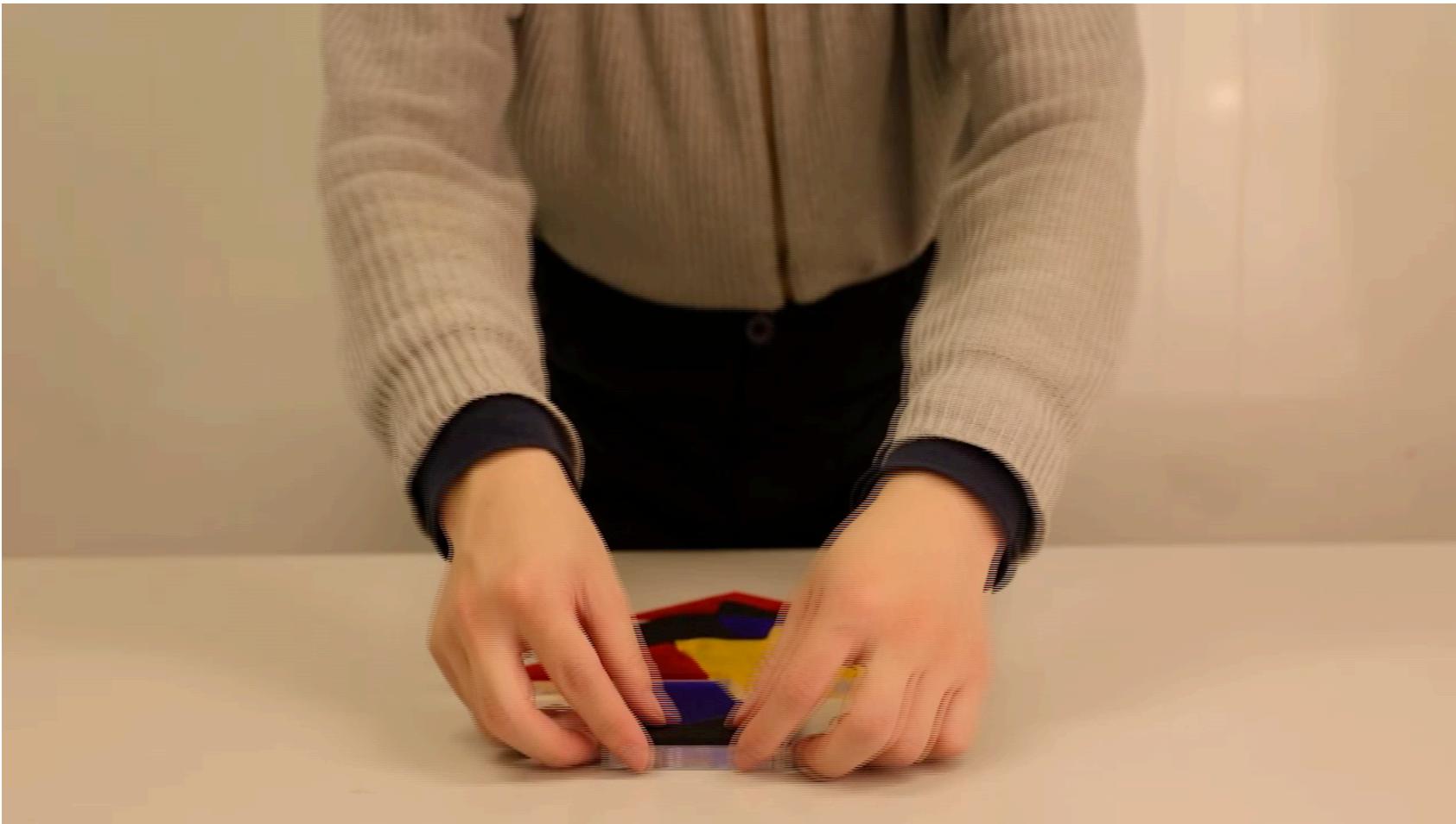


Unstable Input

Motion-based Representation

Stable Output

# Result



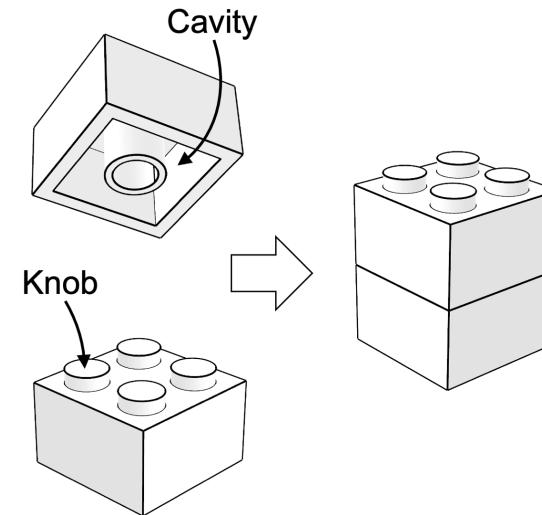
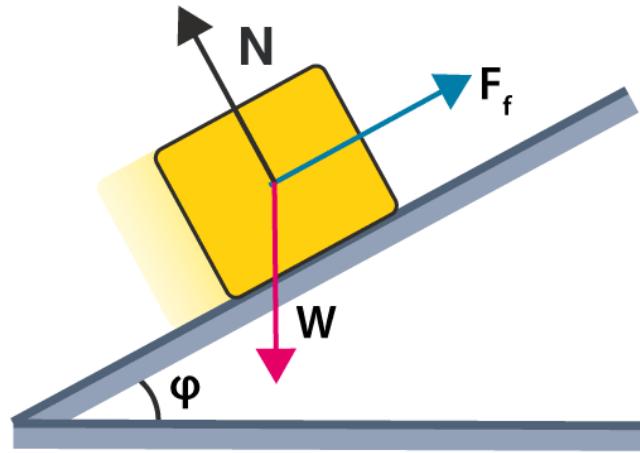
[Wang et al. 2021]

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# Friction

# Friction

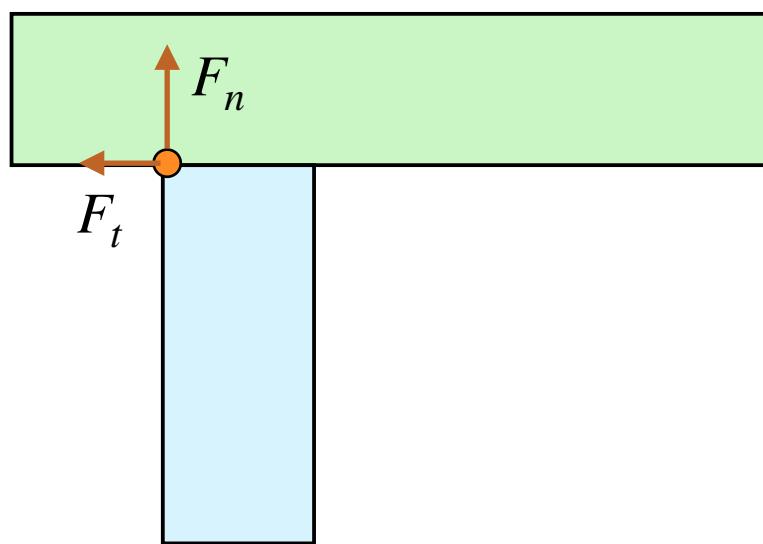
- Friction prevents the relative movement of adjacent parts.
- Many assemblies that use snap joints need friction to stay stable.



Snap Joint

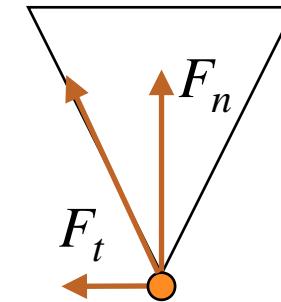
# Coulomb Friction

- The resultant force must be within the friction cone.



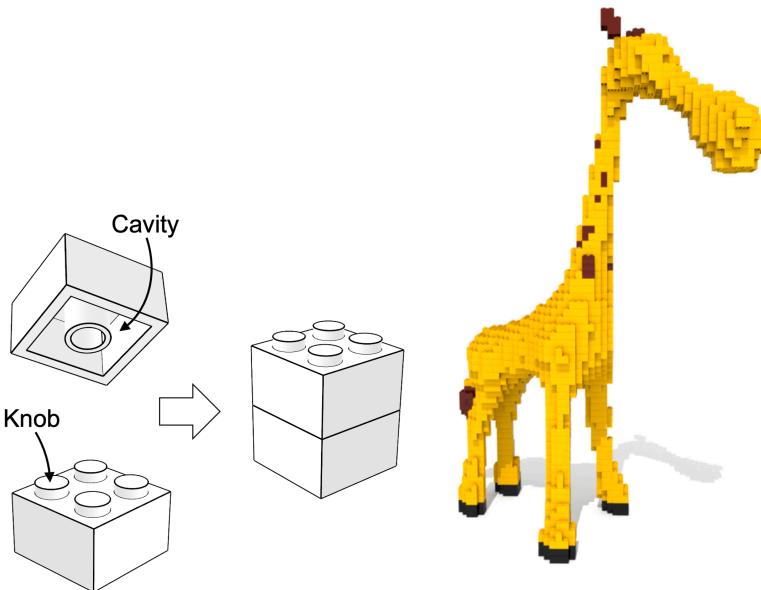
Friction Cone

$$F_t \leq \mu F_n$$

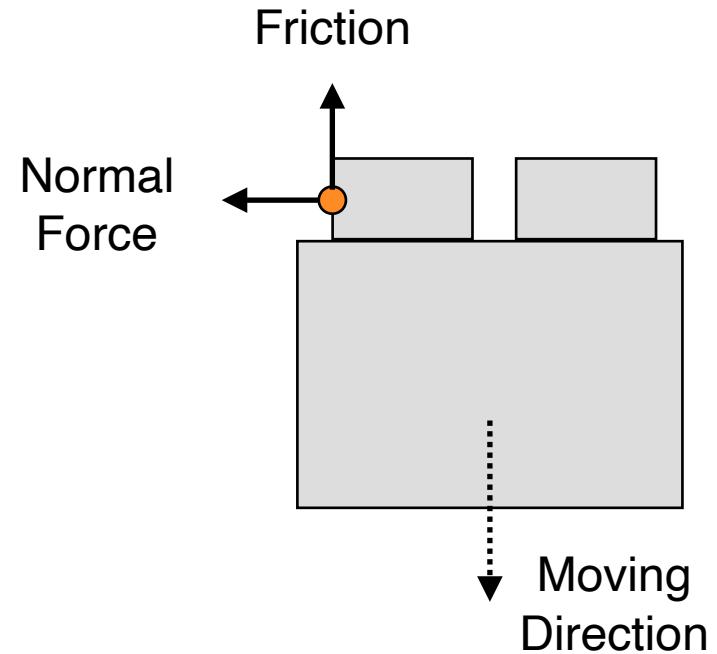


# Friction for LEGOs

- For Legos, the normal forces are constant.
- The friction forces must be within a precomputed range.

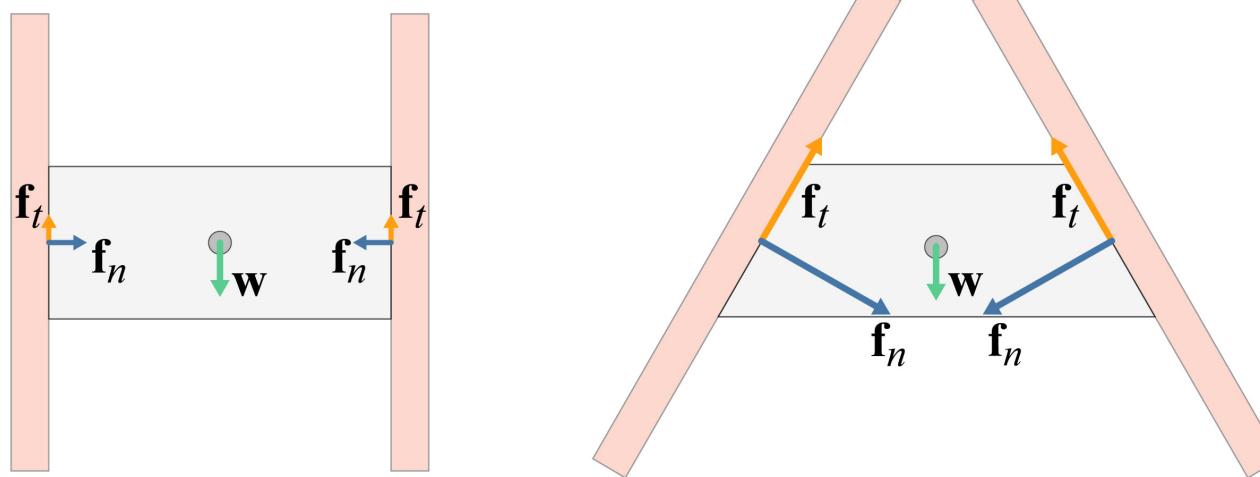


[Luo et al. 2015]



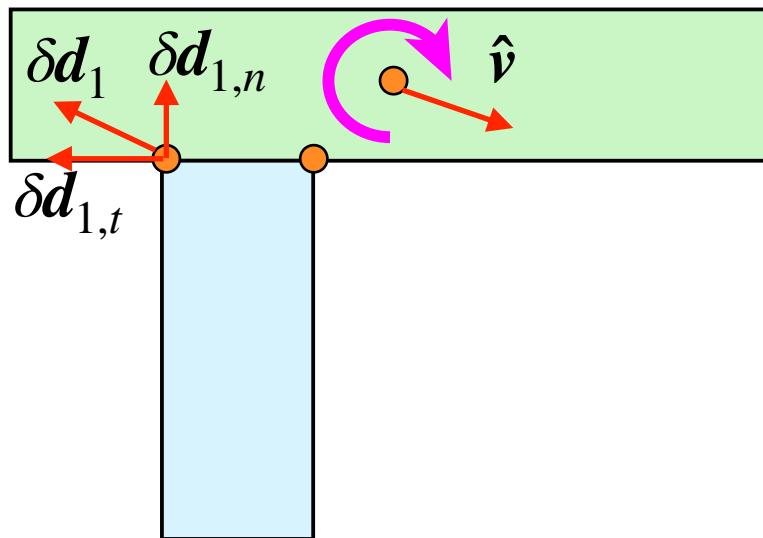
# Limitations of Coulomb Friction

- The Coulomb friction may produce unrealistic force configurations.
- The most well-known failure case is the sliding issue.



# Additional Physical Principles

- Adding more constraints to regulate the friction helps avoid unrealistic cases.



$$\delta d = B_{in} \hat{v}$$

Complementary Condition:

$$\delta d_{1,n} \cdot f_{1,n} = 0$$

Maximum Dissipation

$$f_{1,t} = -\alpha_1 \delta d_{1,t}$$

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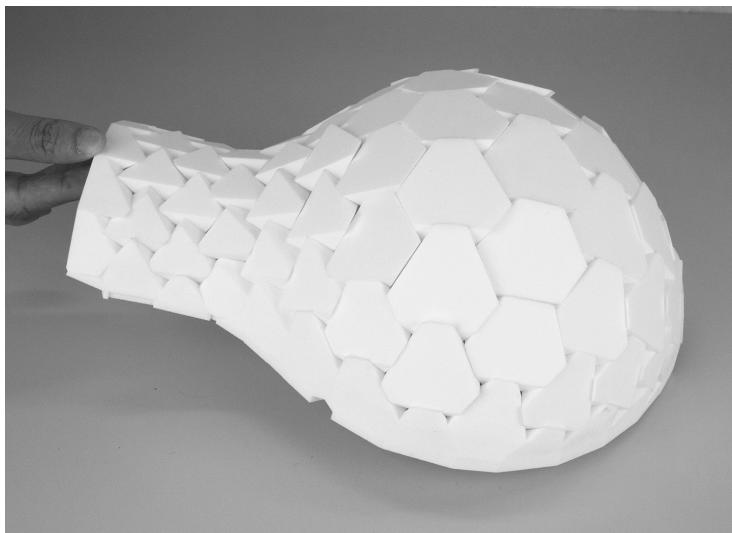
## Part 3: Design for More Types of Stability

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# Lateral Stability

# Lateral Stability

- Assemblies with lateral stability are in equilibrium for a cone of gravity direction.

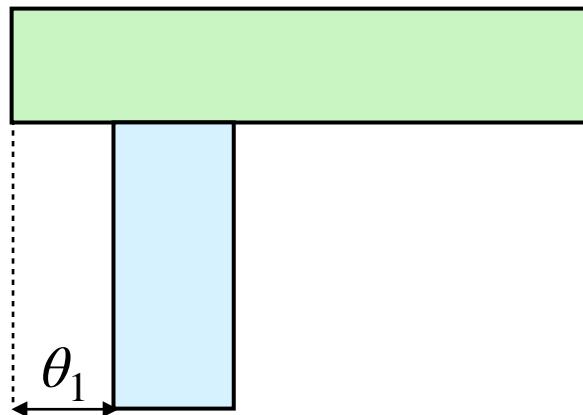


[Wang et al. 2019]

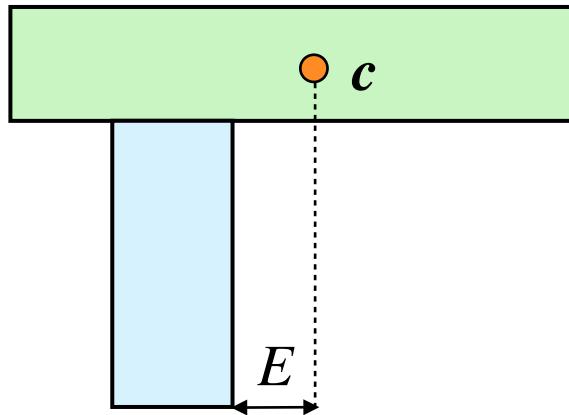


# Recap: Gradient-based Stability Optimization

- Come up with new infeasibility energy for lateral stability.



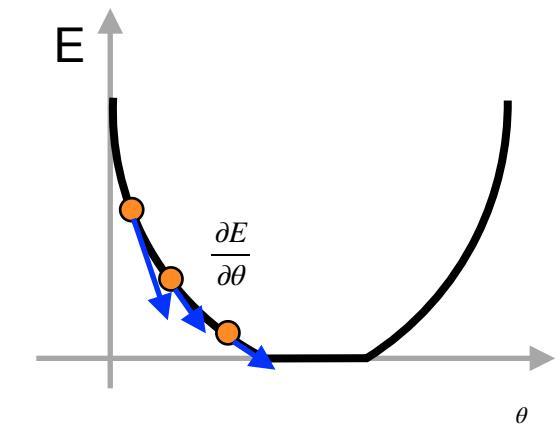
Step 1  
Geometrical Property



Step 2  
Infeasibility Energy

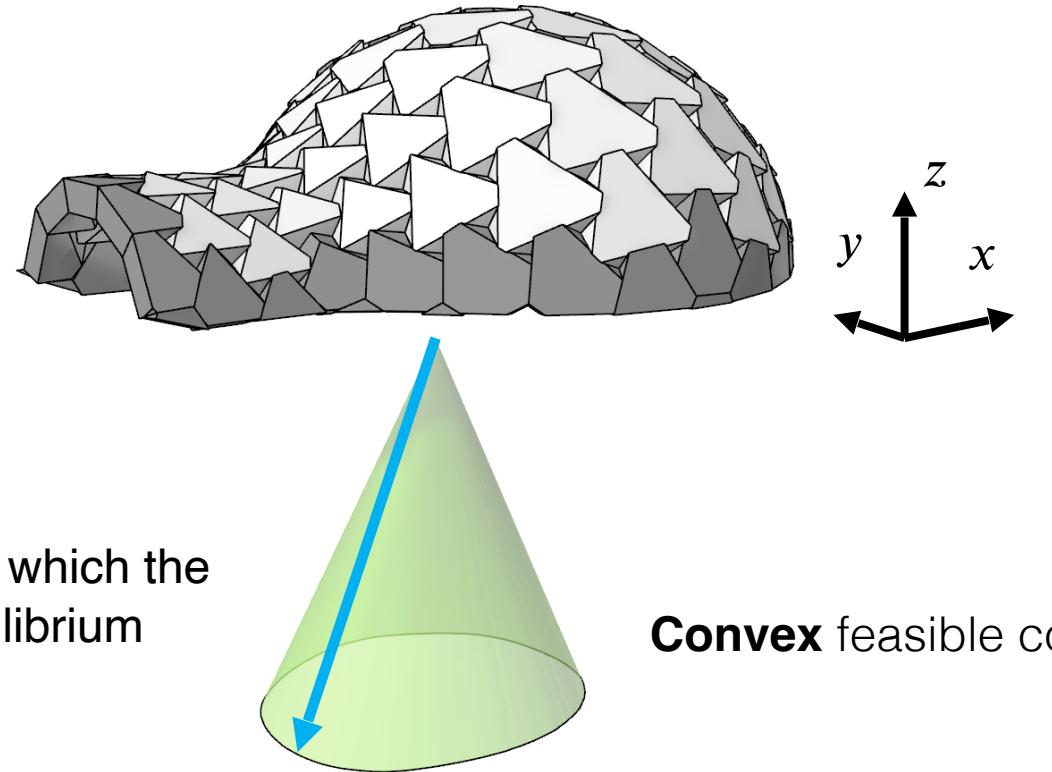
$$\theta \xrightarrow{\partial} c \xrightarrow{\partial} E$$

Step 3  
Sensitivity Analysis

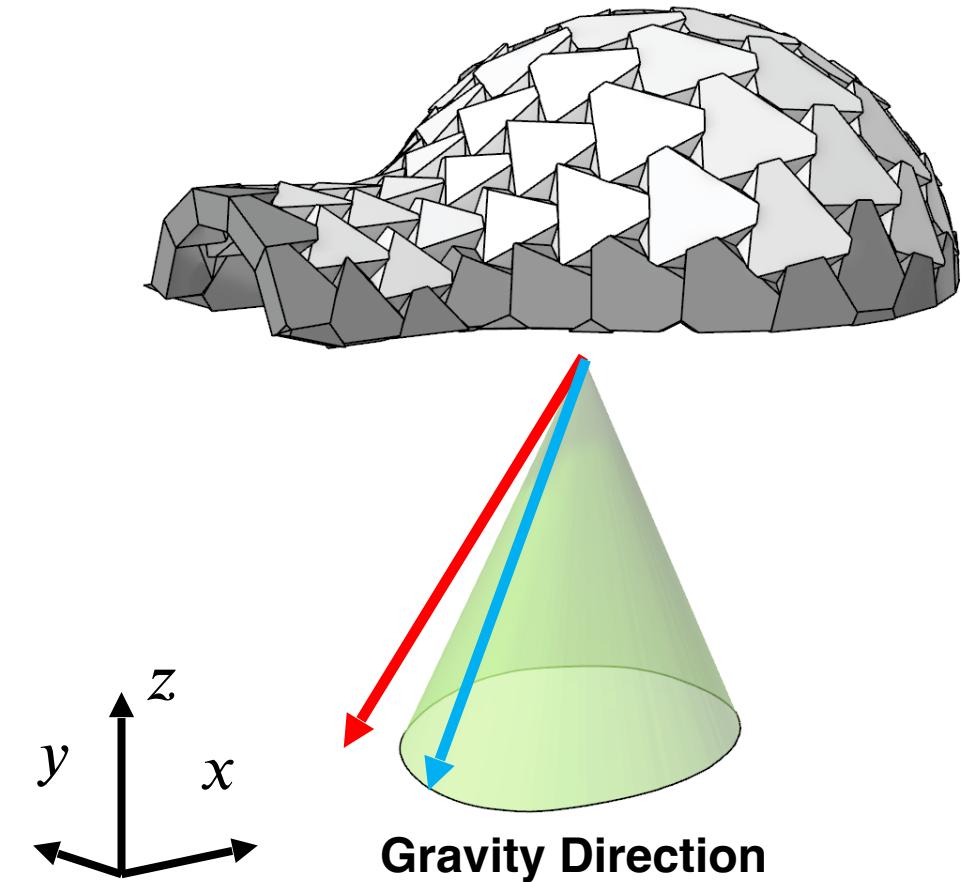


Step 4  
Numerical Optimization

# Feasible Gravitational Cone



# Lateral Infeasibility Measurement

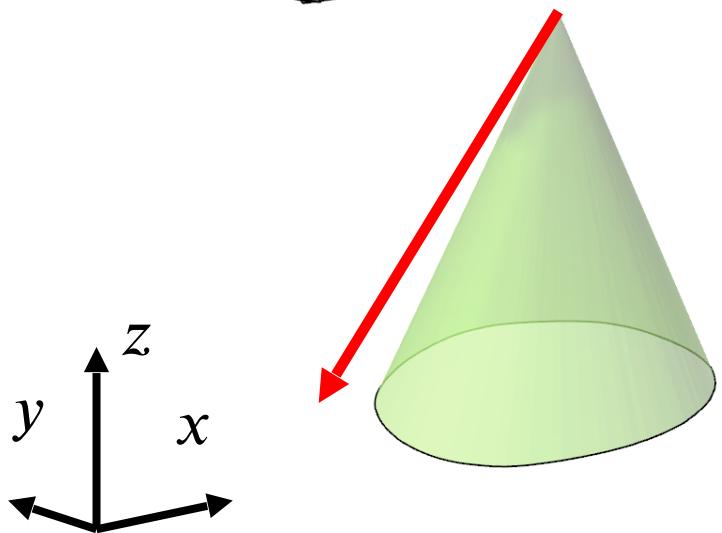
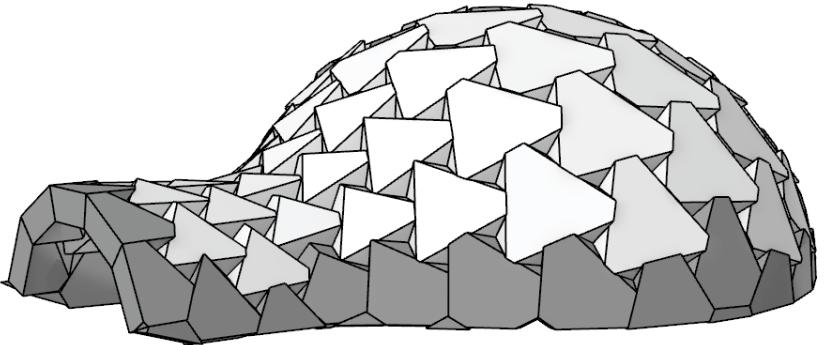


Structural Infeasibility

$$E(\uparrow) = 0$$

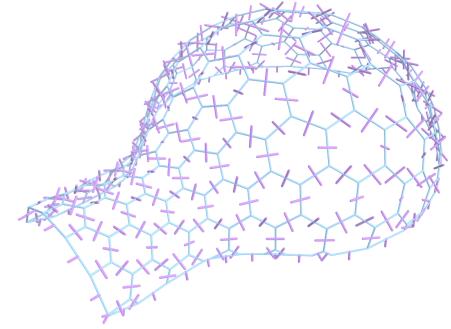
$$E(\downarrow) > 0$$

# Stability Optimization Algorithm

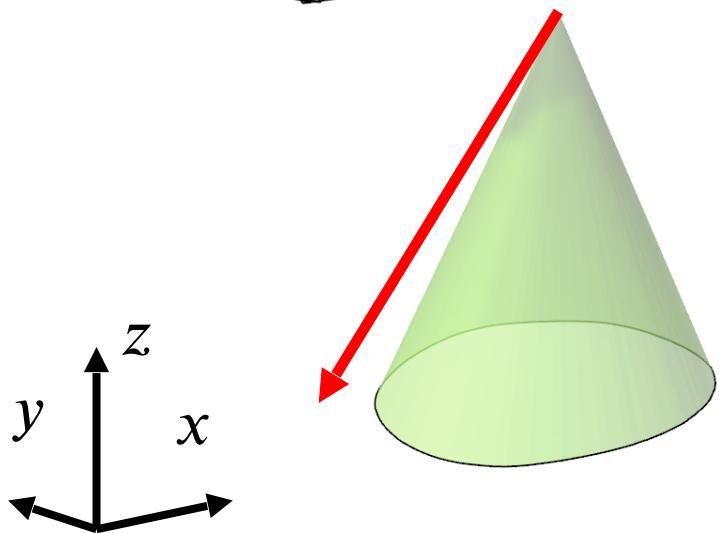
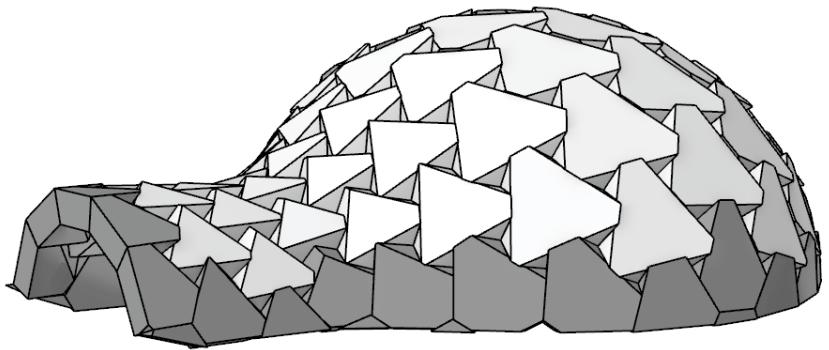


$$\partial E( \quad )$$

with respect to the design parameters



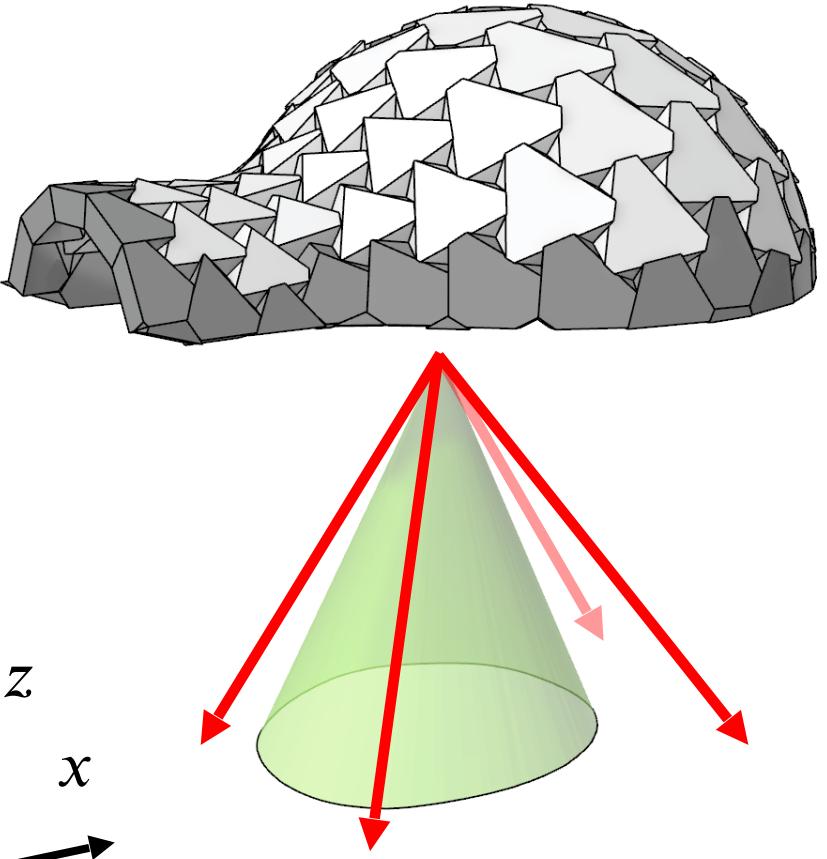
# Stability Optimization Algorithm



$$\min E(\quad)$$

Solved by a **BFGS** Solver

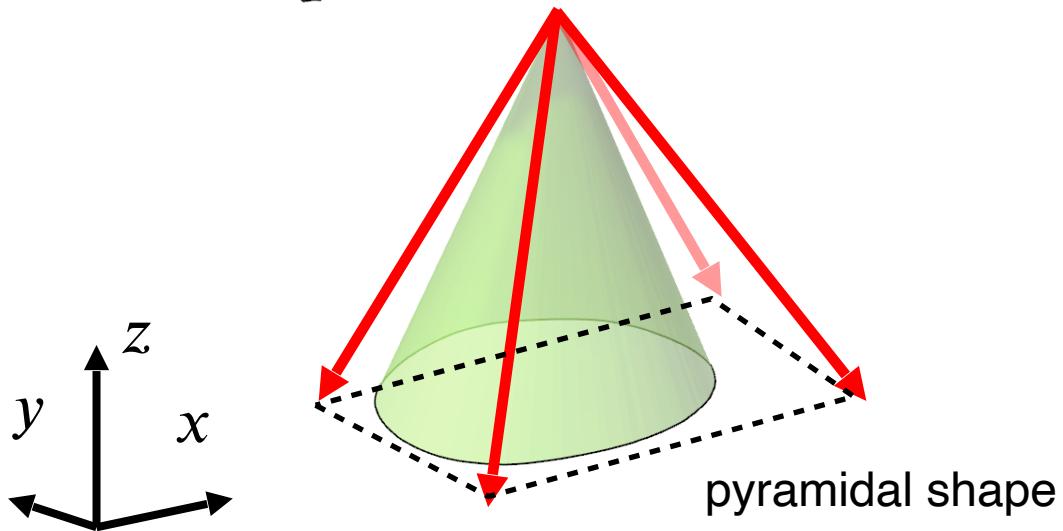
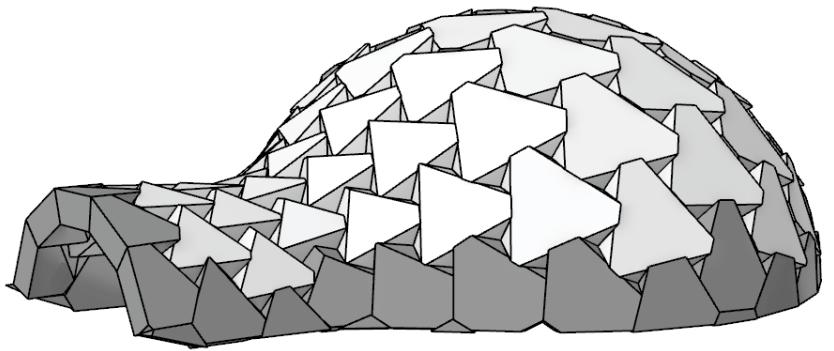
# Lateral Infeasibility Measurement



$$\min E(\swarrow) + E(\searrow) + E(\nwarrow) + E(\nearrow)$$

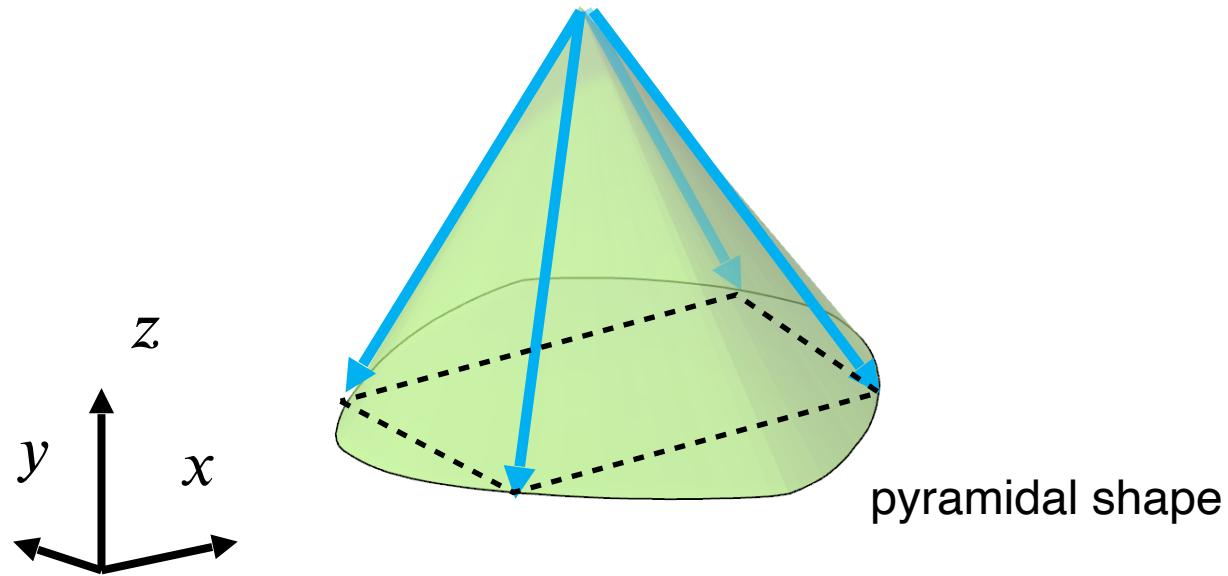
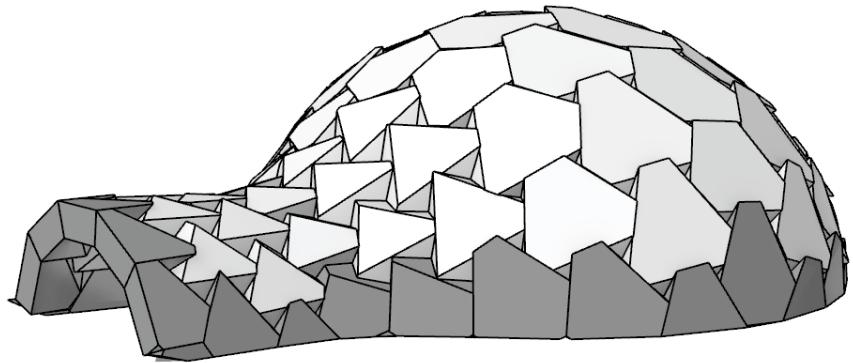
Contact Area  $\geq$  User defined value

# Lateral Infeasibility Measurement



*Due to the convexity of the feasible cone*

# Lateral Infeasibility Measurement



*The new feasible cone  
will cover the pyramidal shape*

# Result

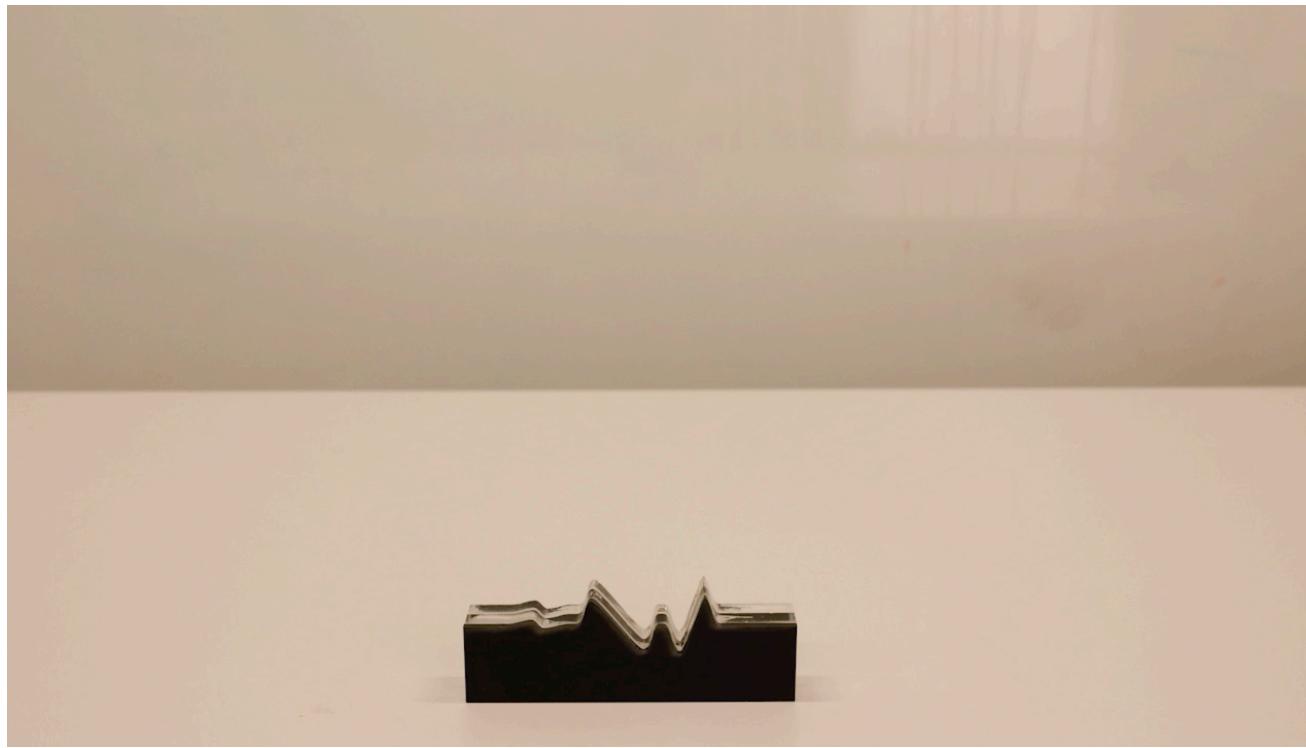


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# Scaffold-free Assembly

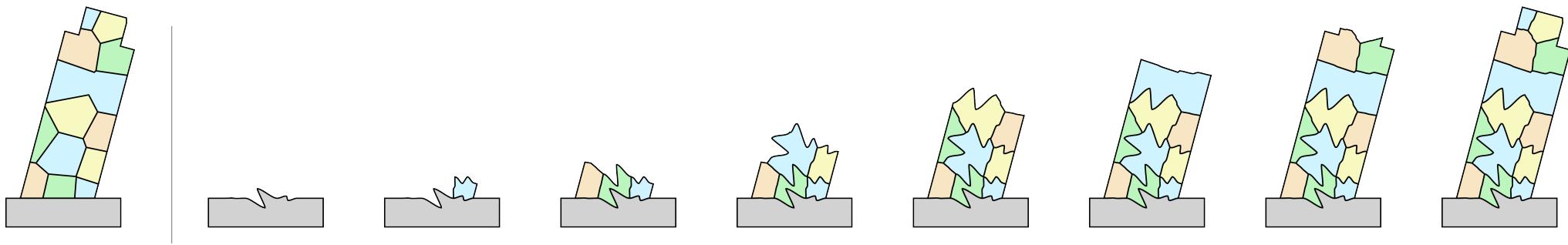
# Scaffold-free Assembly

- Making the assembling process stable.



# Infeasibility Energy

- The infeasibility energy is the summation of all the infeasibility energy of the structure at each assembling stage.



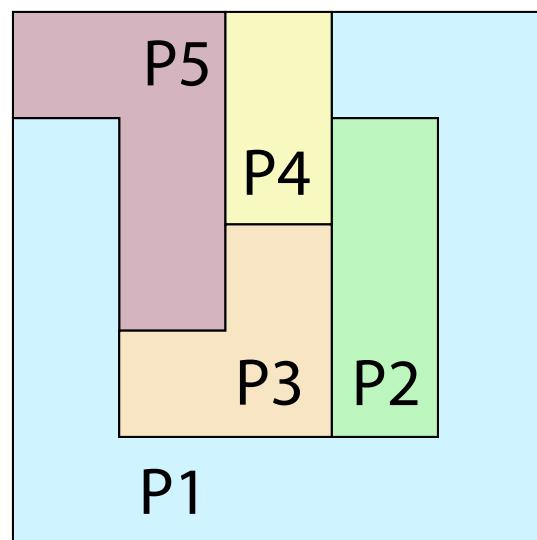
$$E = E_1 + E_2 + E_3 + E_4 + E_5 + E_6 + E_7 + E_8$$

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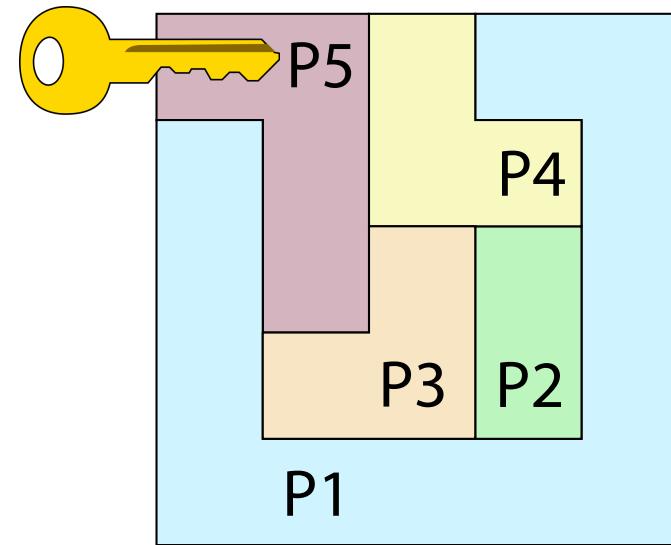
# Globally Interlocking Assemblies

# Recap: Globally Interlocking

Once the key and a part of the reset are fixed, no parts can be taken out from the assembly.



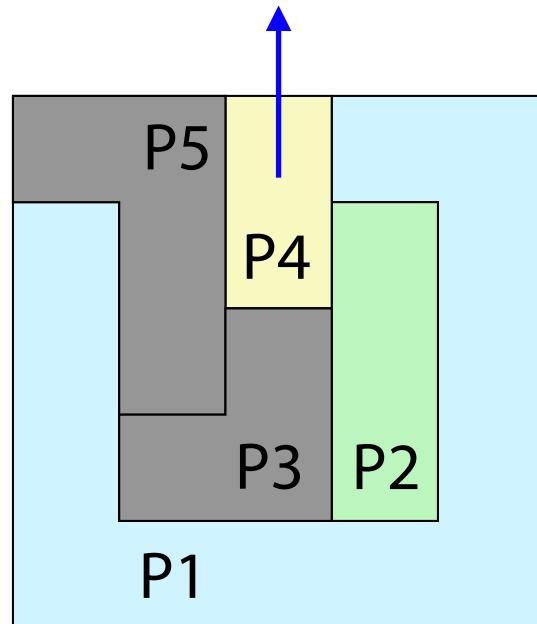
Non Interlocking



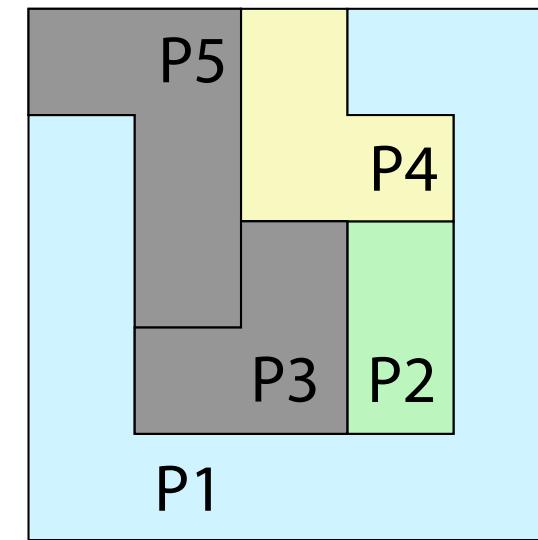
Interlocking

# Recap: Globally Interlocking

Once the key and a part of the reset are fixed, no parts can be taken out from the assembly.



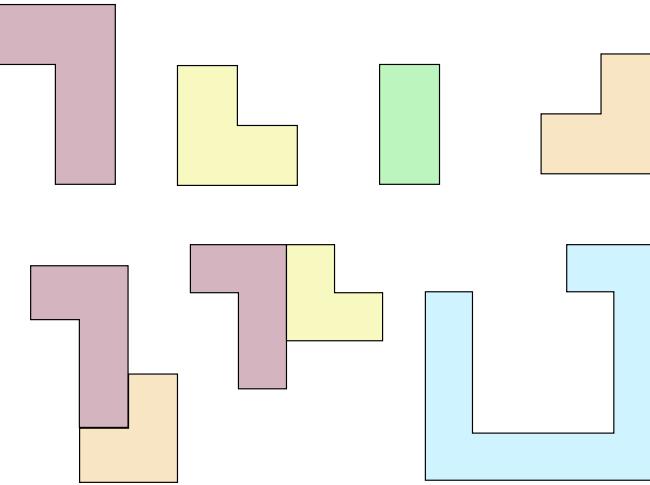
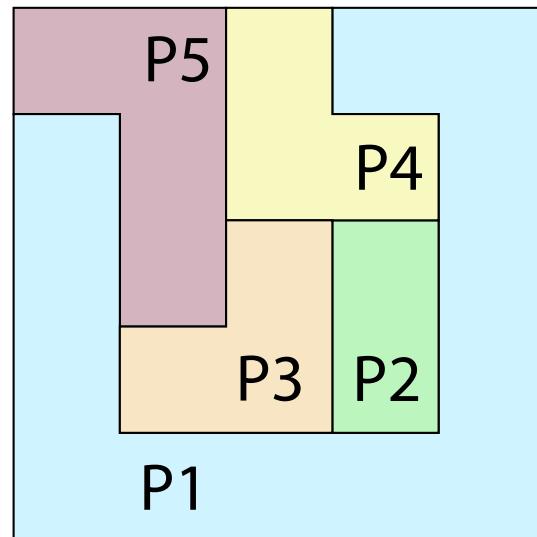
Non Interlocking



Interlocking

# Recap: Classic Interlocking Test

Classic method examines every subset of parts, which has **exponential time complexity**.

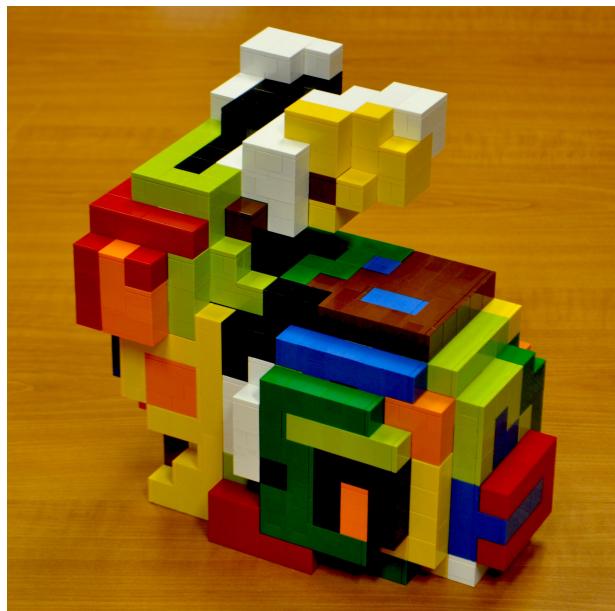


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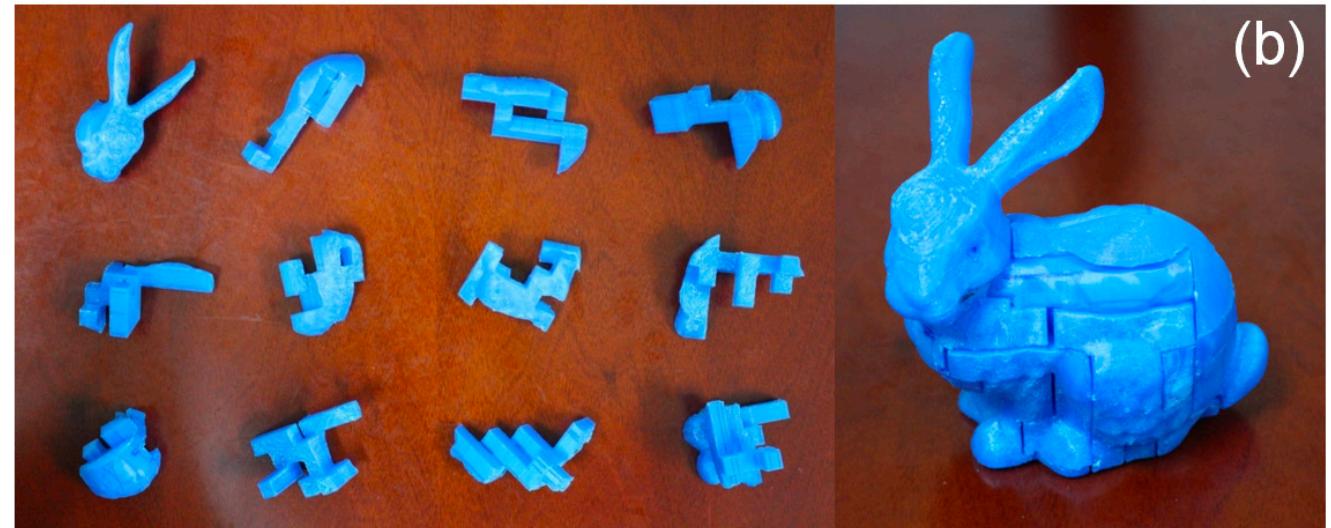
[Song et al. 2012]

# Shape Decomposition

- When the input is a target shape, computational design of interlocking assemblies can be formulated as a shape decomposition problem.



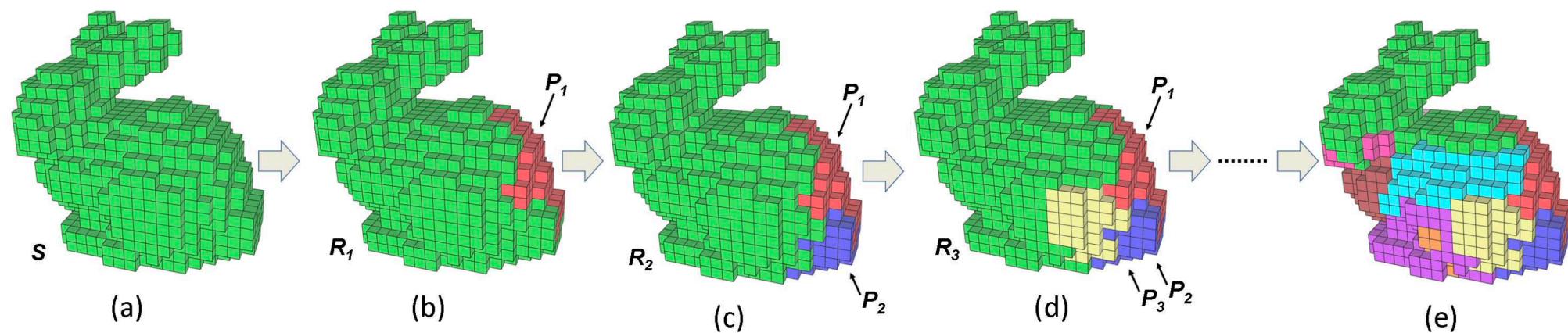
[Song et al. 2012]



[Song et al. 2015]

# Shape Decomposition

- When the input is a target shape, computational design of interlocking assemblies can be formulated as a shape decomposition problem.



[Song et al. 2012]

# Joint Planning

- When the input is a set of initial parts without joints, designing interlocking assemblies can be formulated as a joint planning problem.



[Fu et al. 2015]



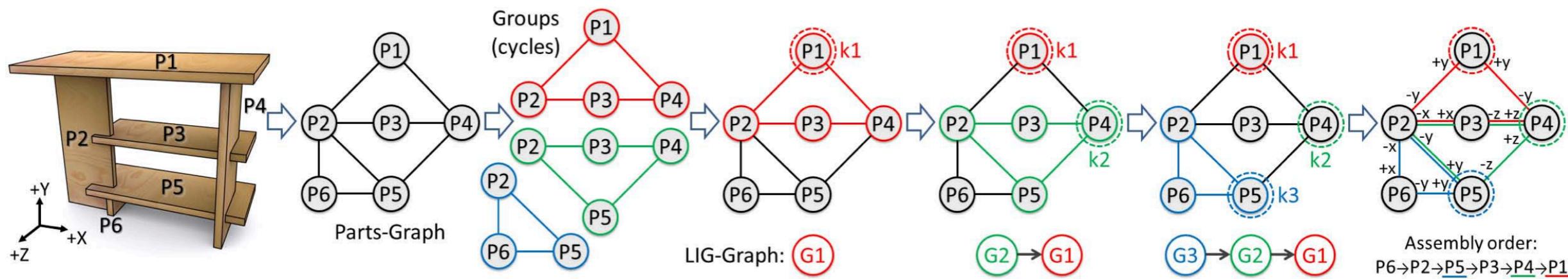
[Song et al. 2016]



[Yao et al. 2017]

# Joint Planning

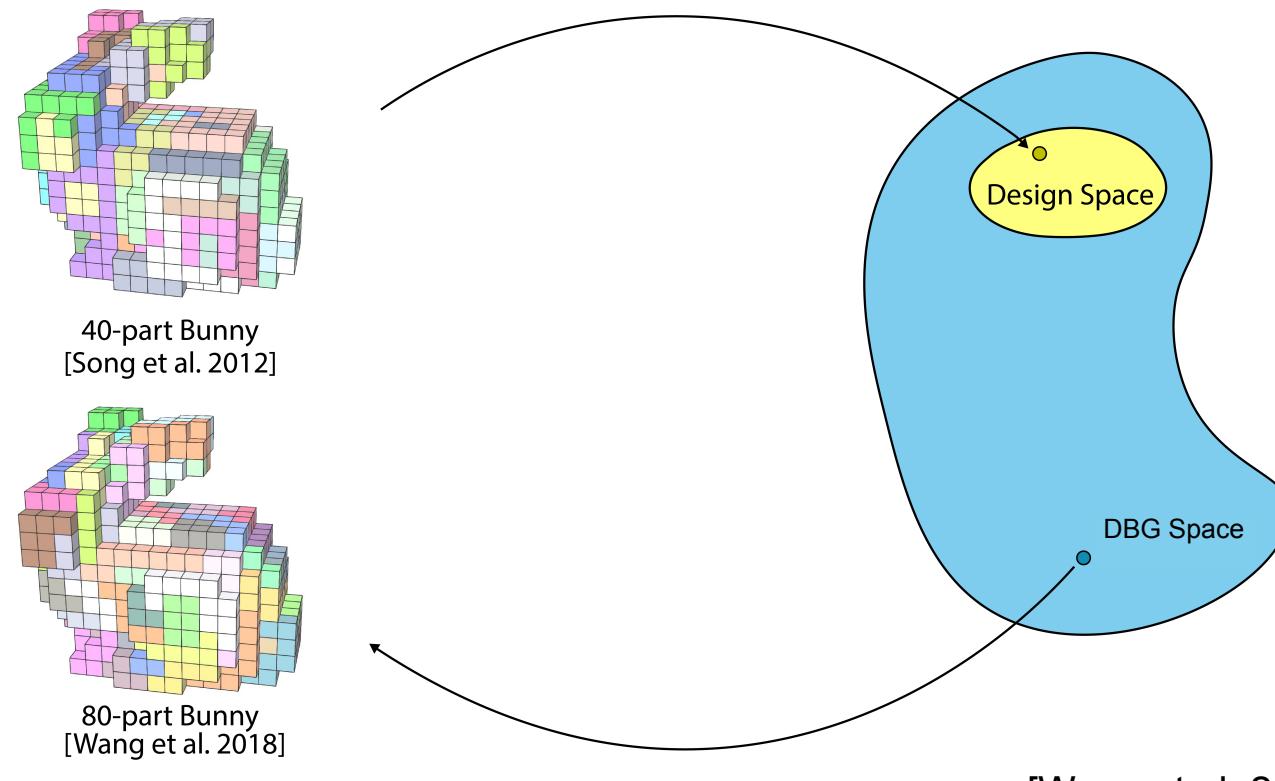
- Fu et al. computed an interlocking joint configuration following the LIG-based approach.



[Fu et al. 2015]

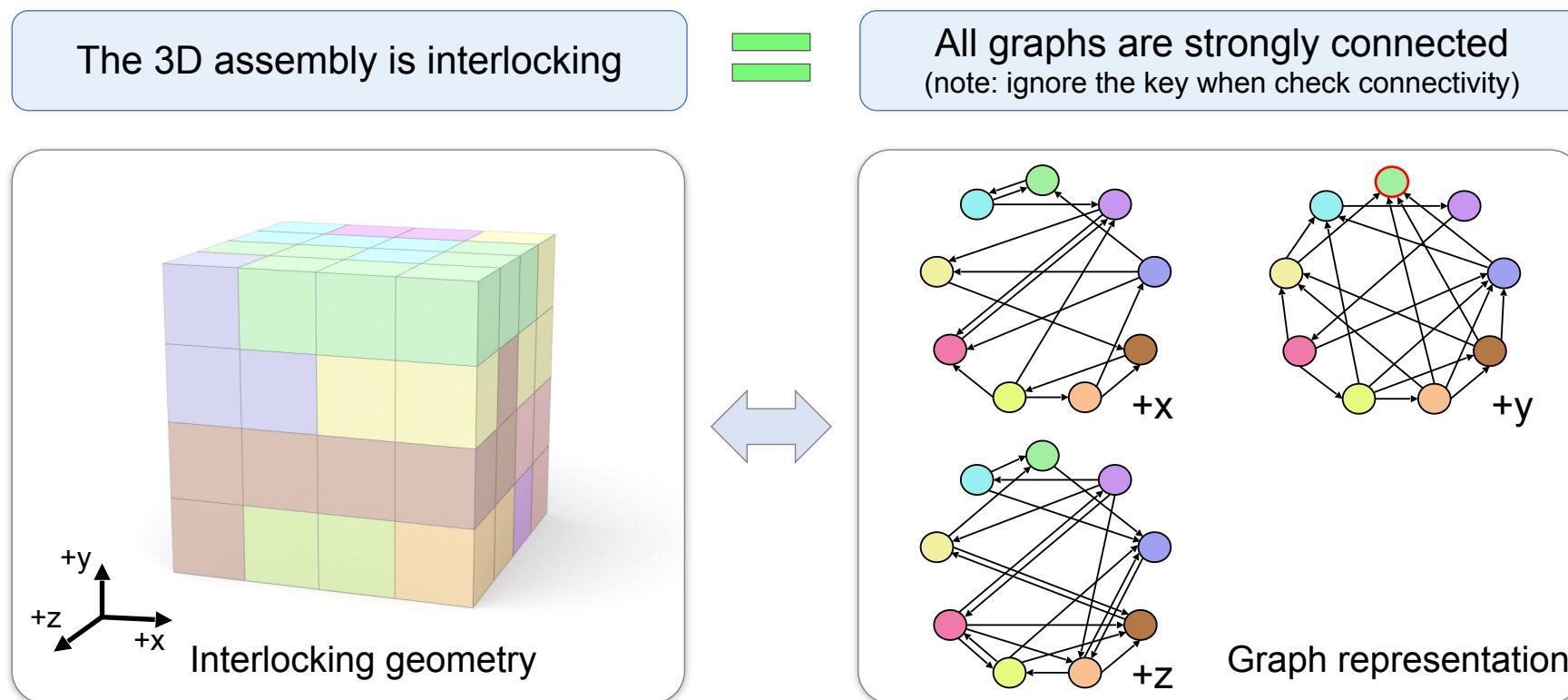
# DBG-based Interlocking Design

- The DBG approach allow exploring the full search space of interlocking configurations.

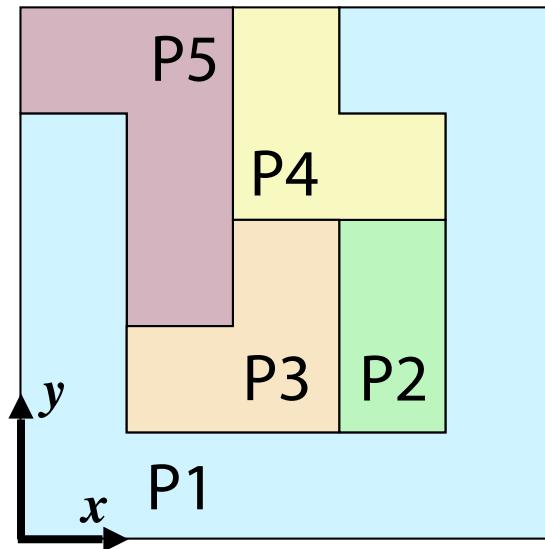


# DBG-based Interlocking Design

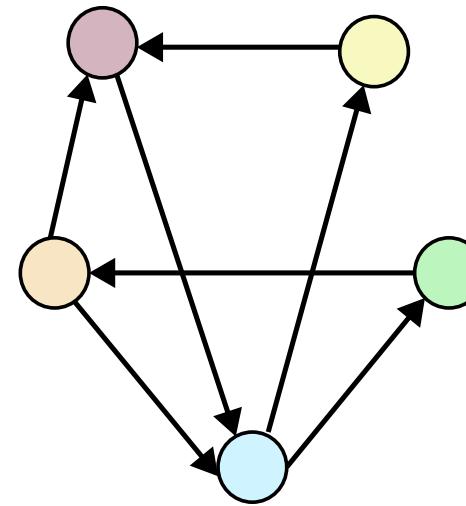
- Wang et al. use the base DBG to test and design interlocking assemblies.



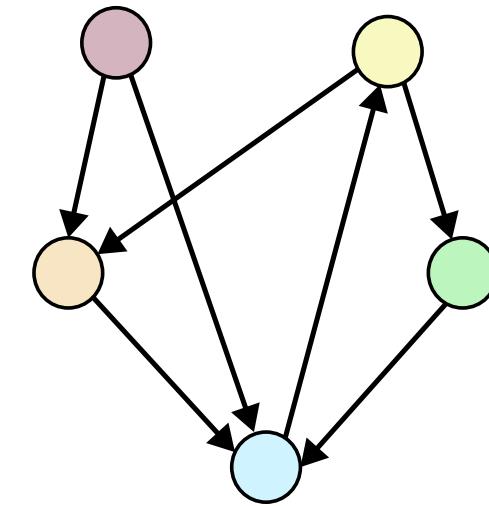
# Directional Blocking Graphs



$G(+x)$



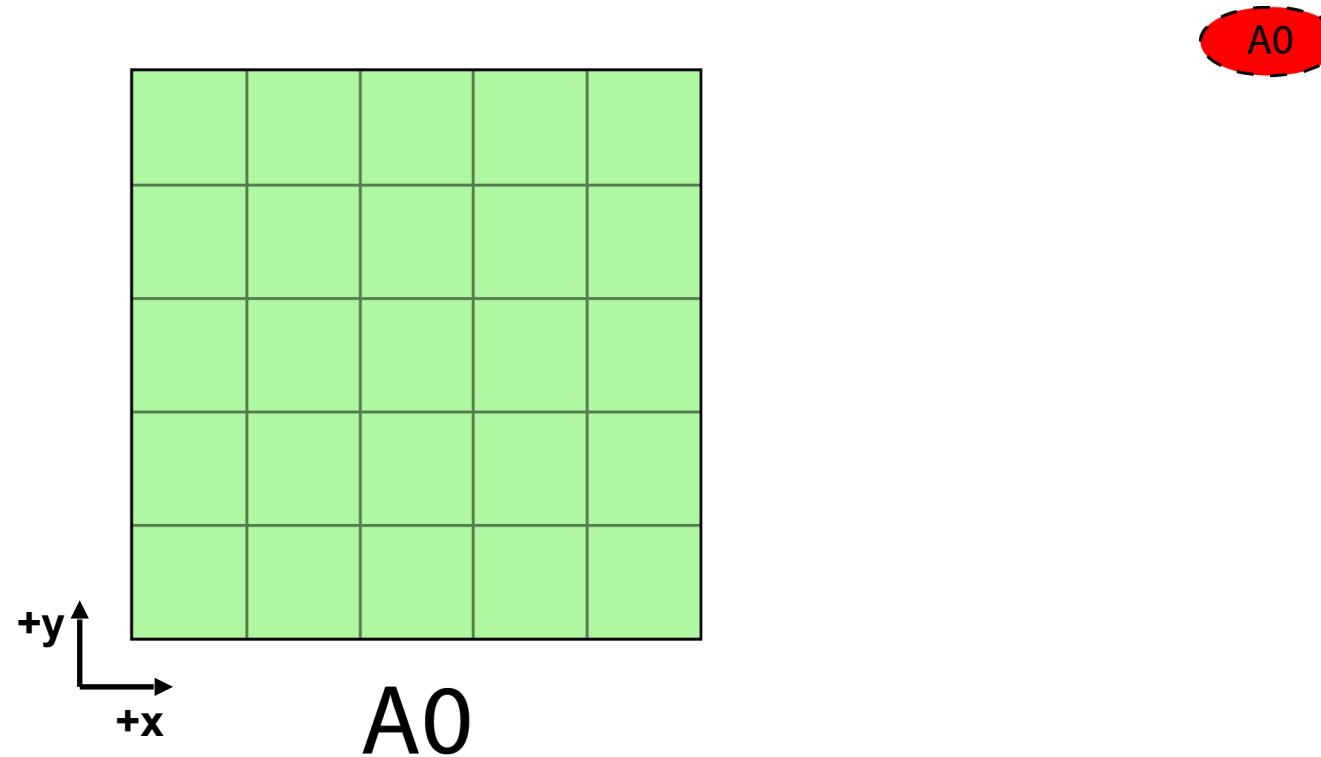
$G(+y)$



An assembly is global Interlocking when its directional blocking graphs are **strongly connected** except the key

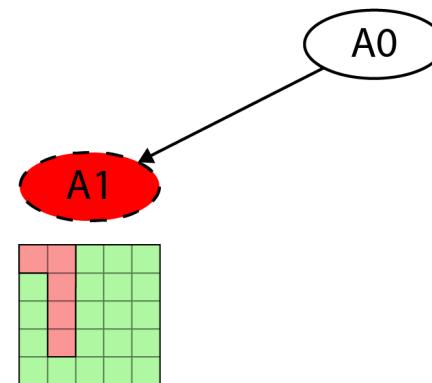
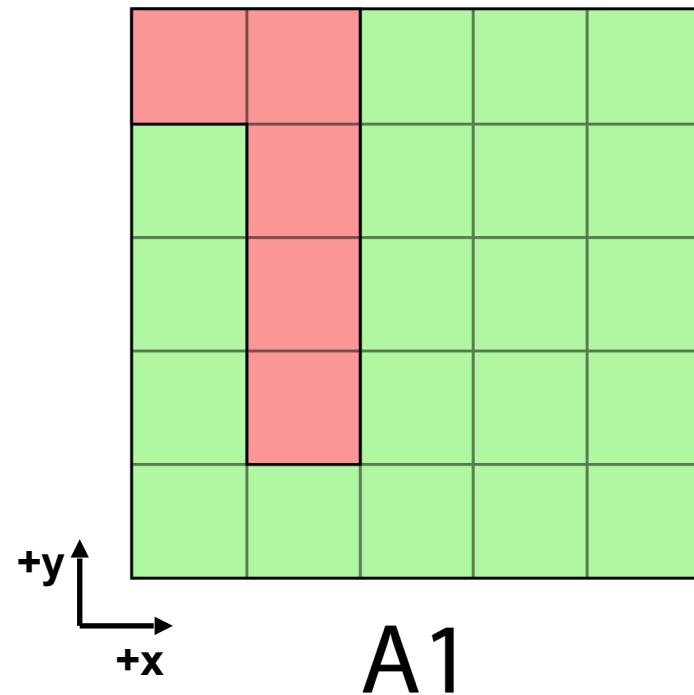
# Interactive Design Framework

Example: design a 5-part interlocking assembly by partitioning a 5x5 square.

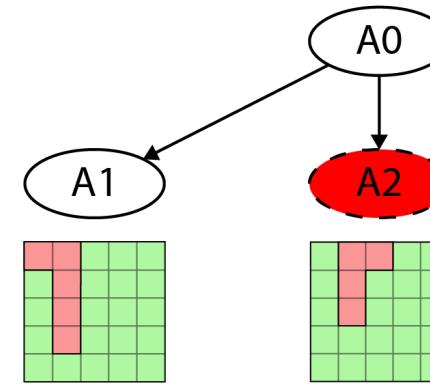
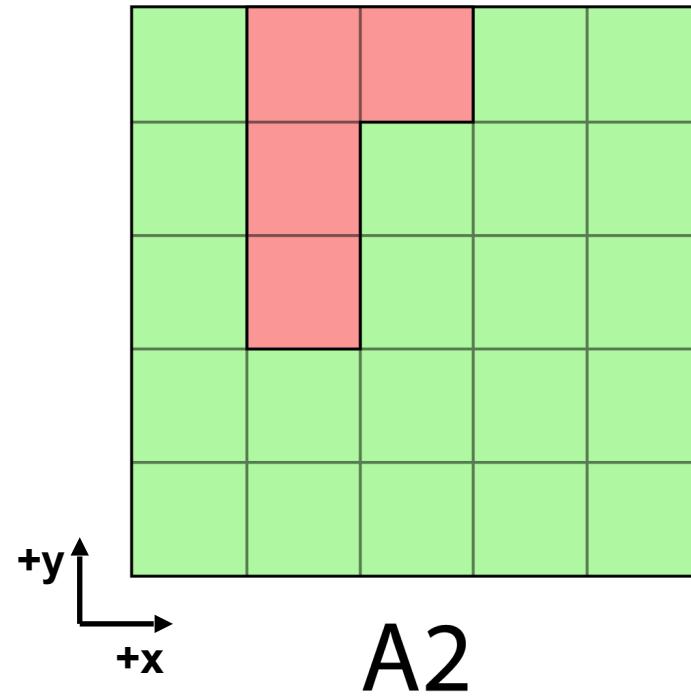


# Interactive Design Framework

First, construct the key part.

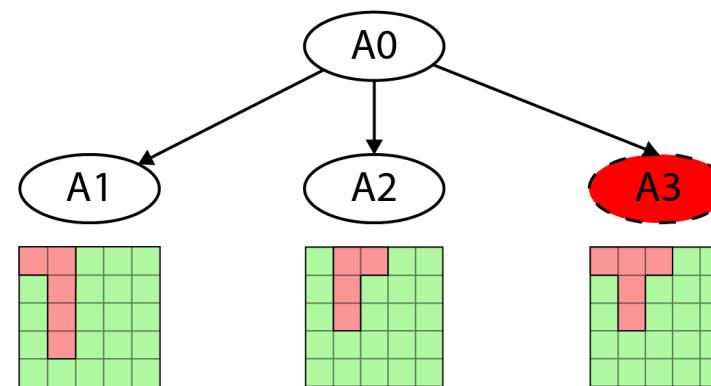
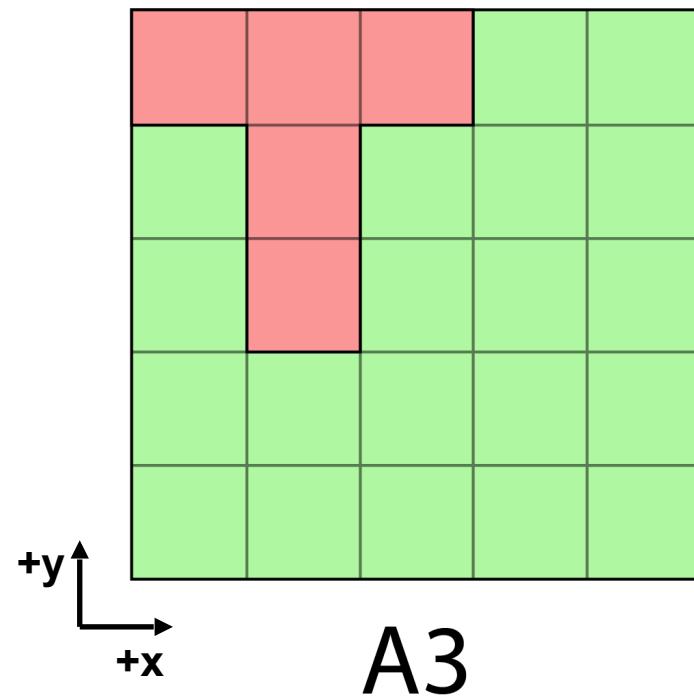


# Interactive Design Framework



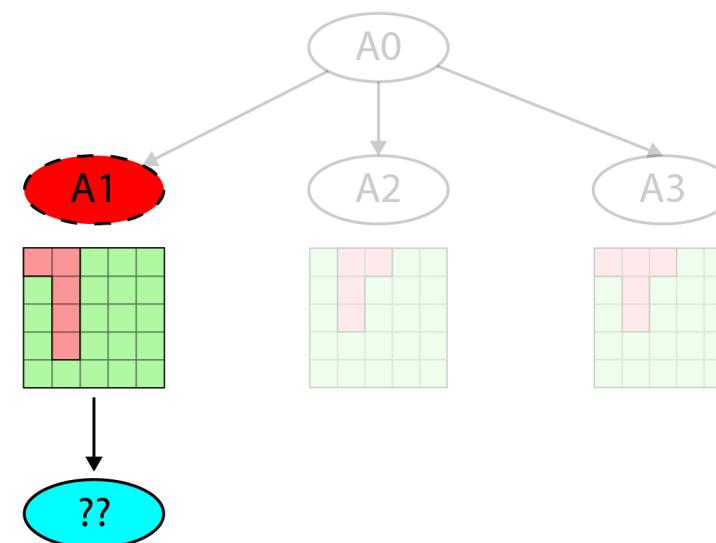
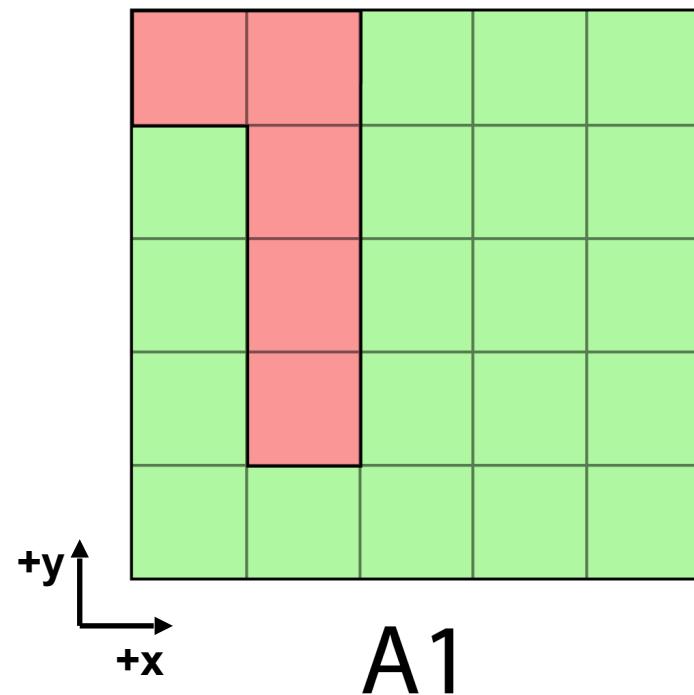
# Interactive Design Framework

There are many possibilities and we only select a few candidates.



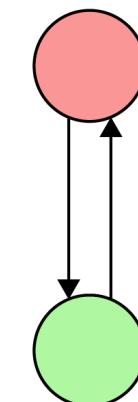
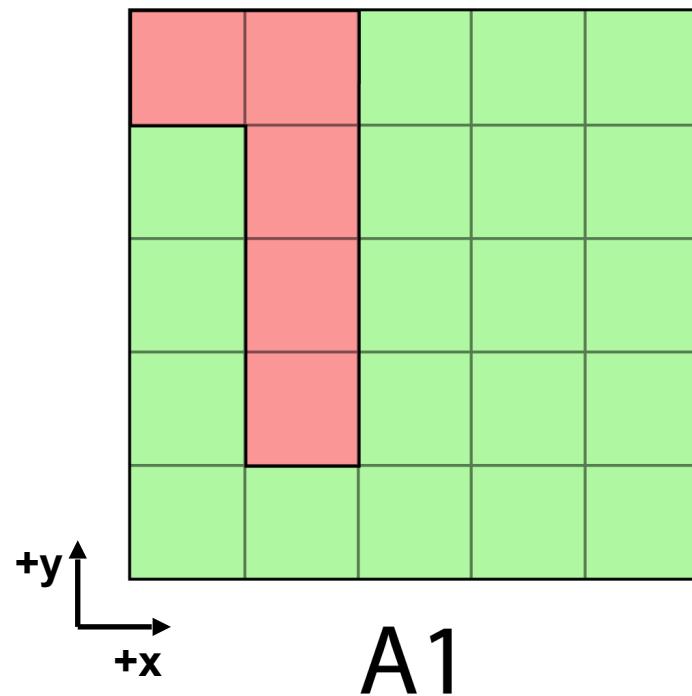
# Interactive Design Framework

How to design a 3-part interlocking assembly by partitioning A1?

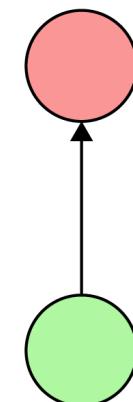


# Graph Design

Construct the base directional blocking graphs for A1.



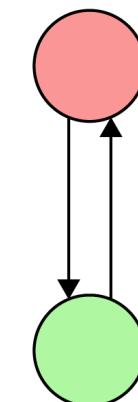
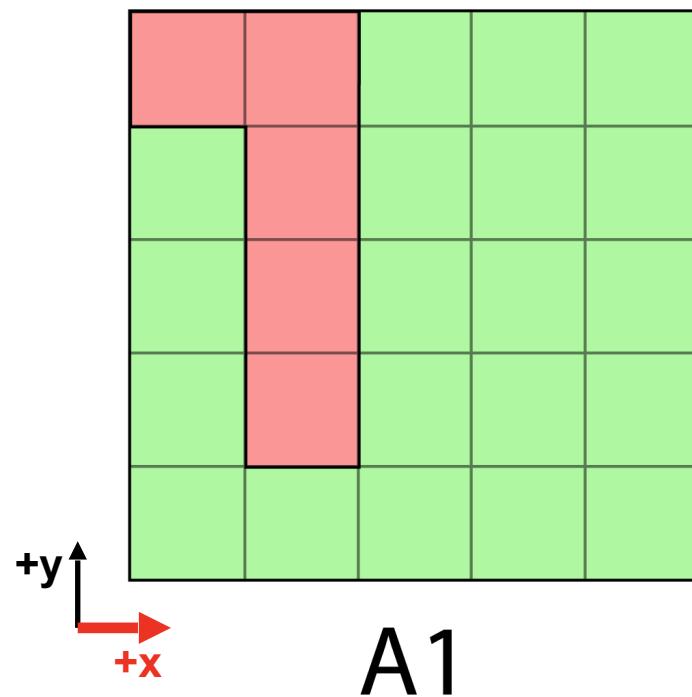
**G(+X)**



**G(+Y)**

# Graph Design

Construct the base directional blocking graphs for A1.



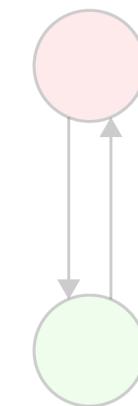
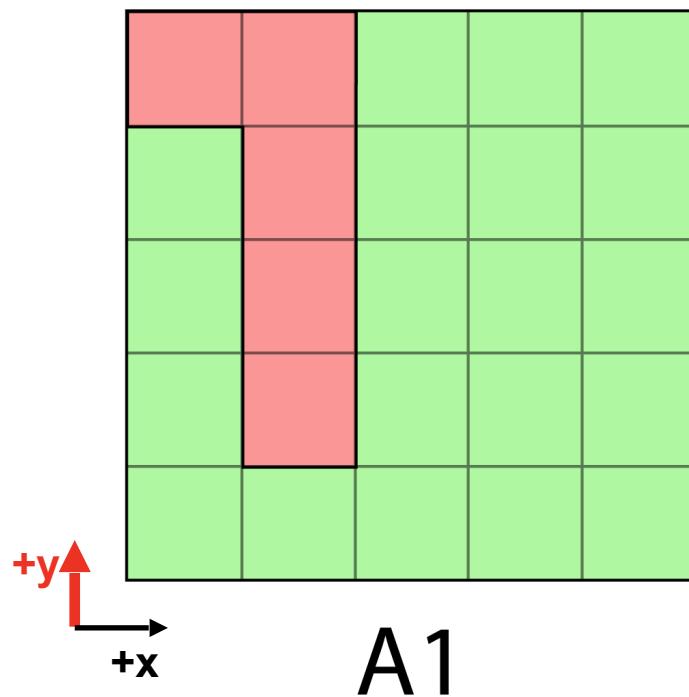
**G(+X)**



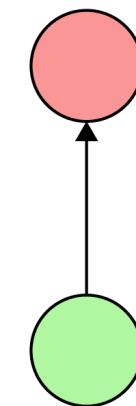
**G(+Y)**

# Graph Design

Construct the base directional blocking graphs for A1.



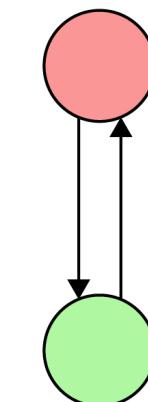
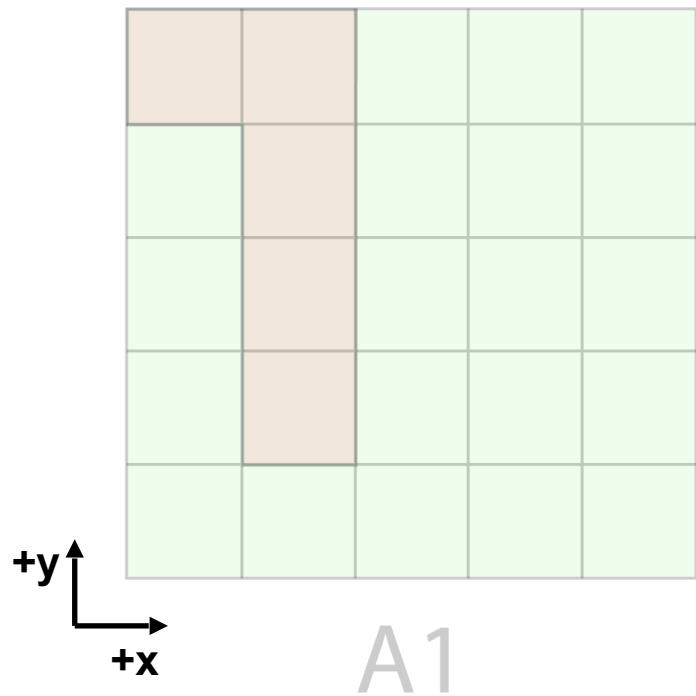
**G(+X)**



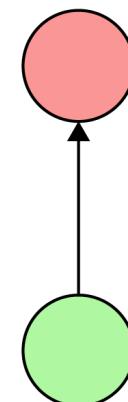
**G(+Y)**

# Graph Design

Ignore geometry of the assembly.



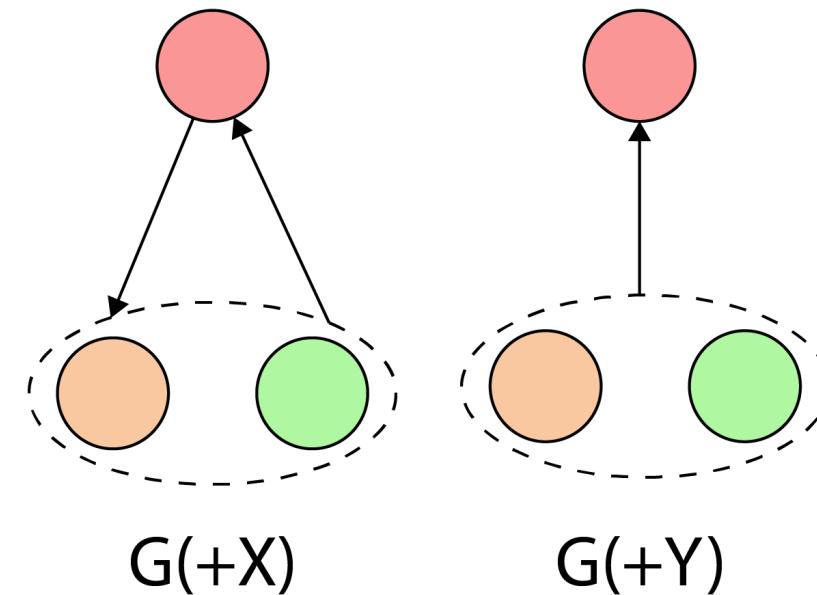
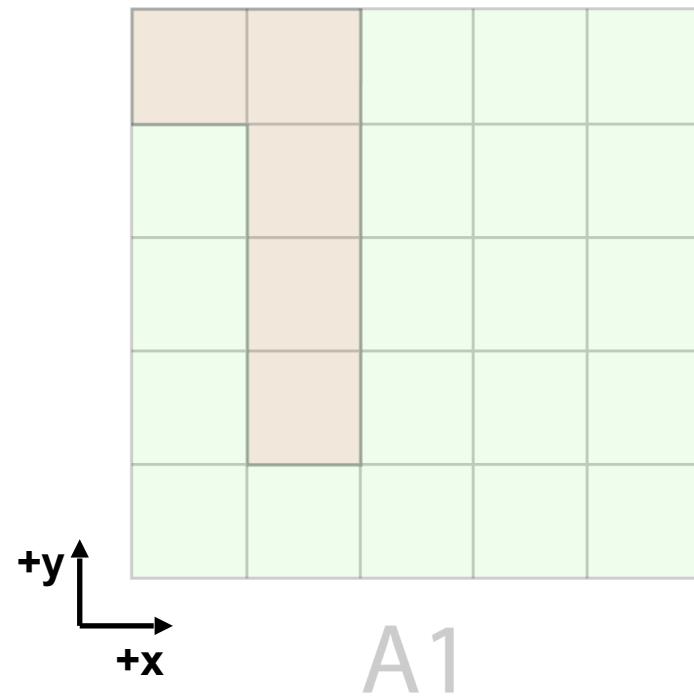
$G(+X)$



$G(+Y)$

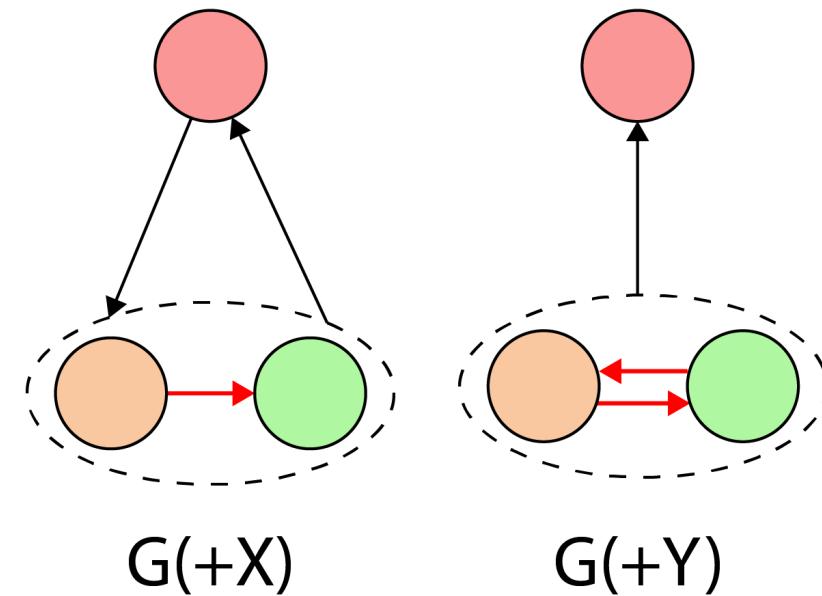
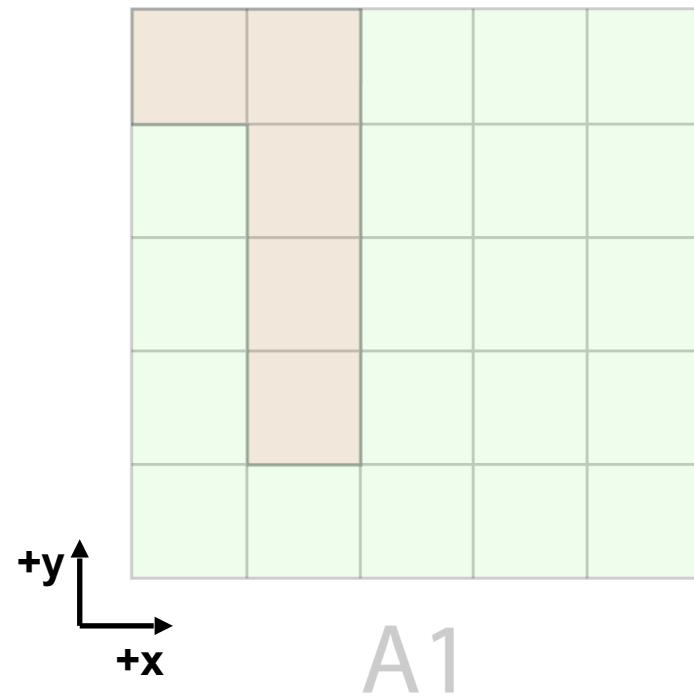
# Graph Design

How to make graphs strongly connected?



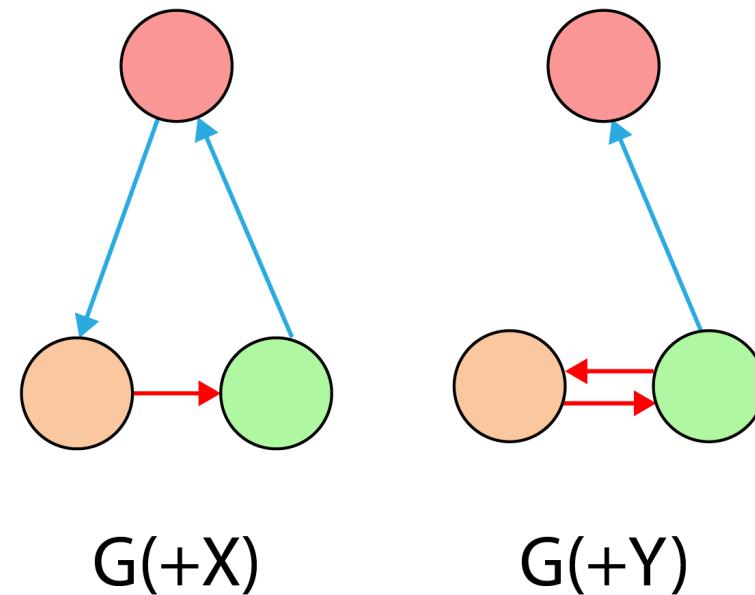
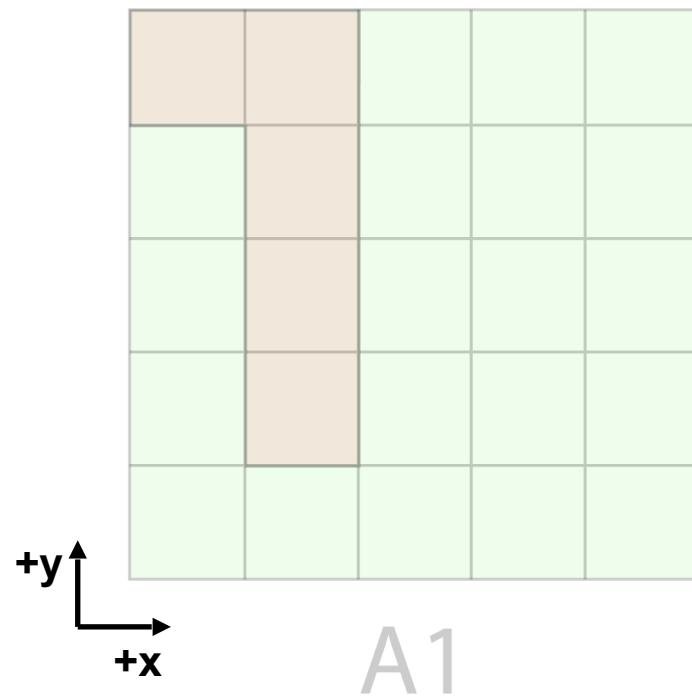
# Graph Design

First choose the internal blocking relation between two split nodes.



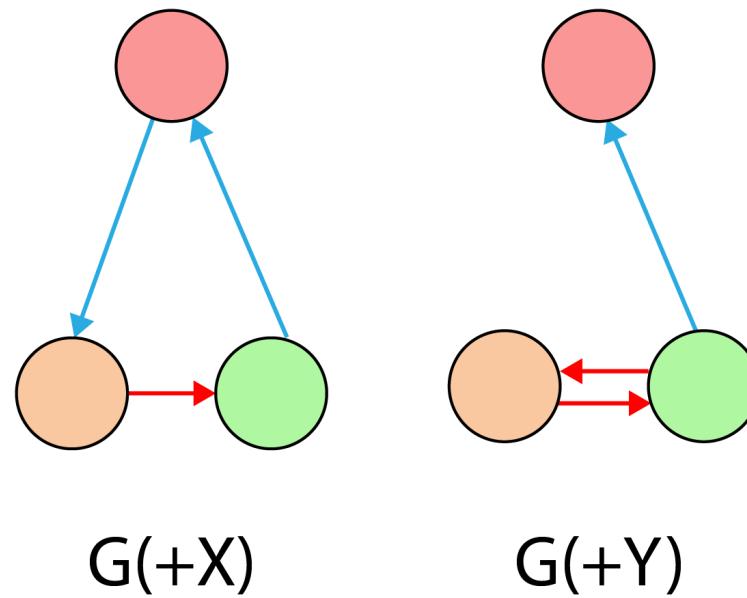
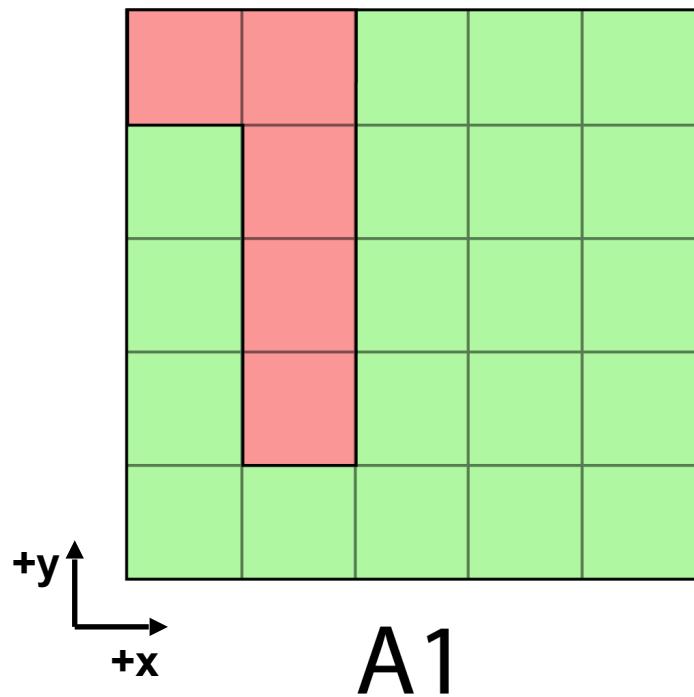
# Graph Design

Then distribute the external blocking relations to the two split nodes.



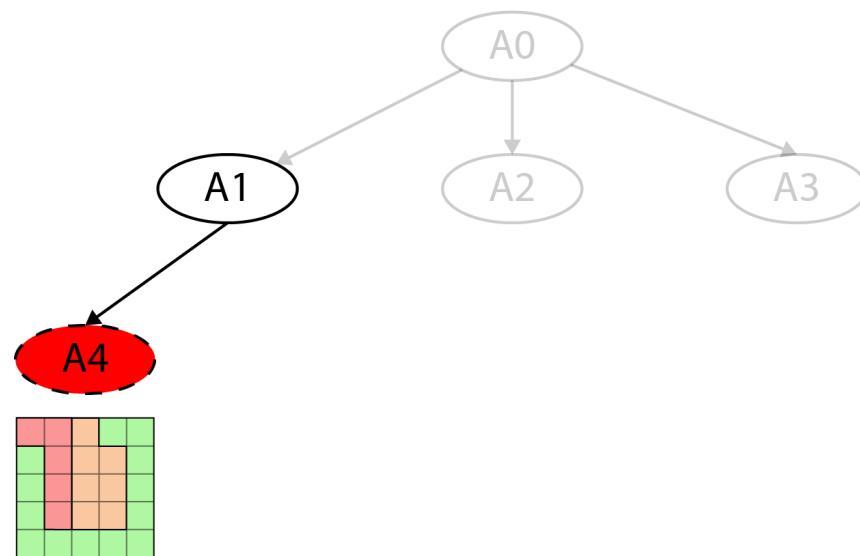
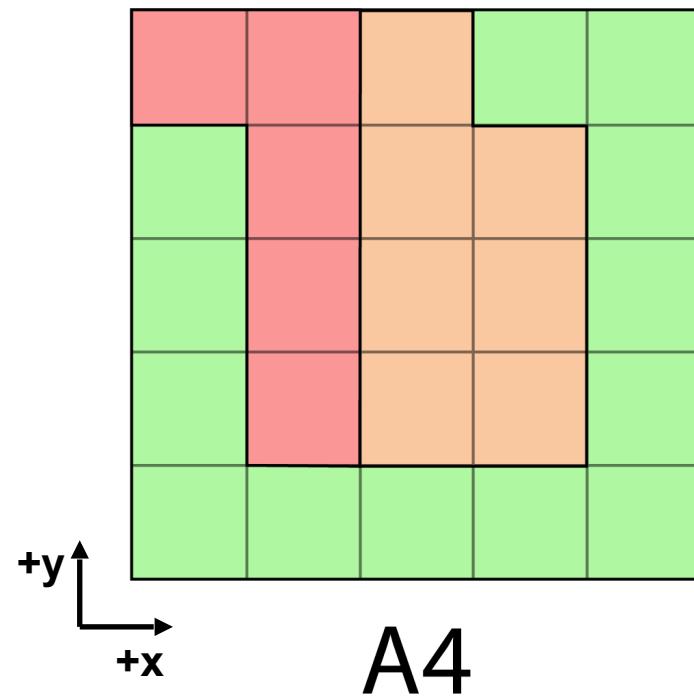
# Geometry Realization

Find geometry corresponding to the designed base DBGs.



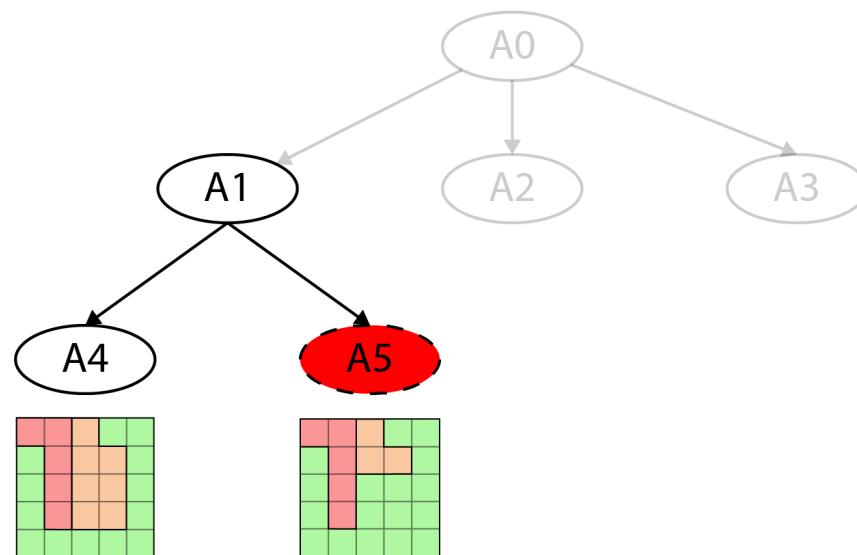
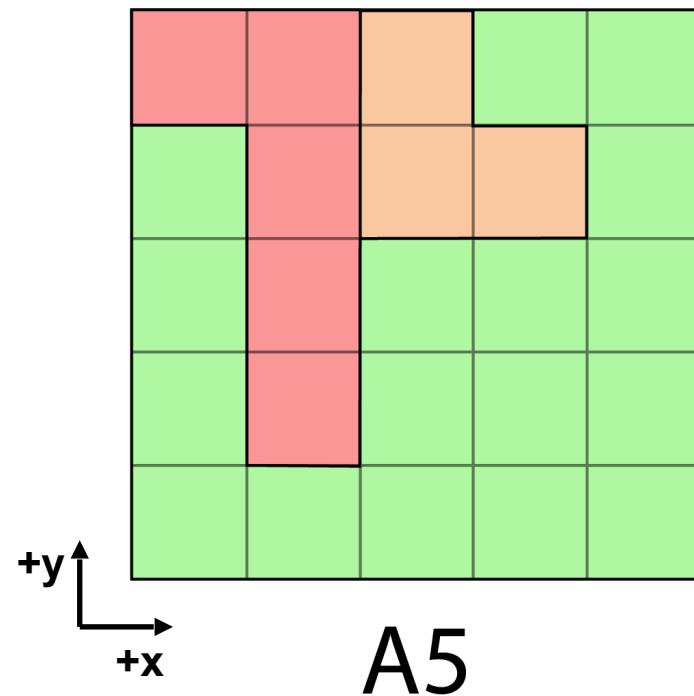
# Interactive Design Framework

Generate candidates of a 3-part interlocking assembly.



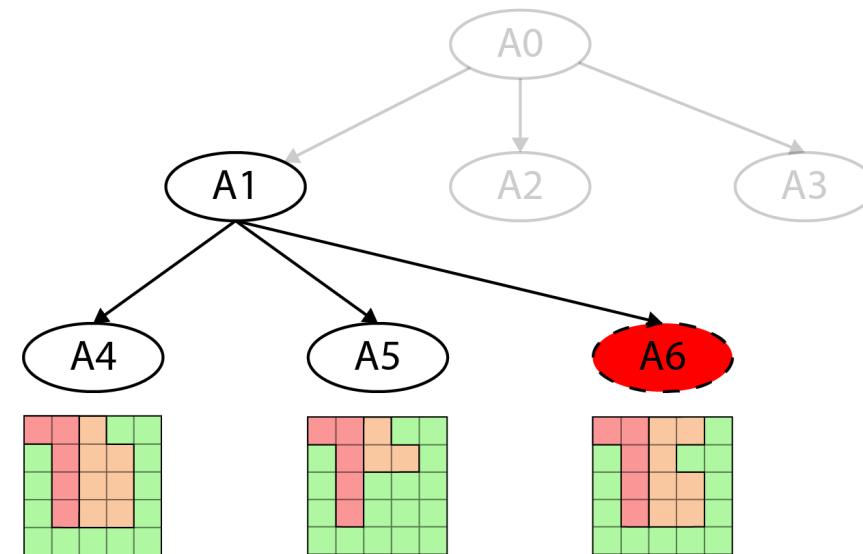
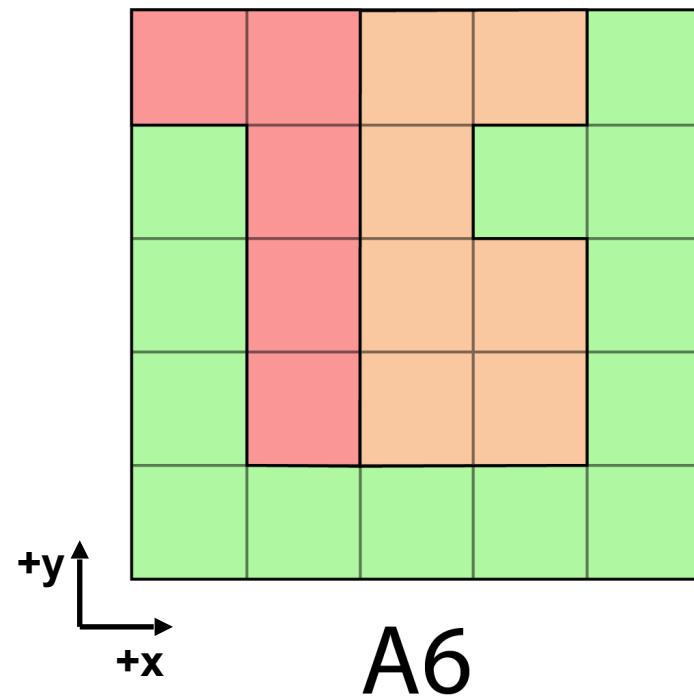
# Interactive Design Framework

Generate candidates of a 3-part interlocking assembly.



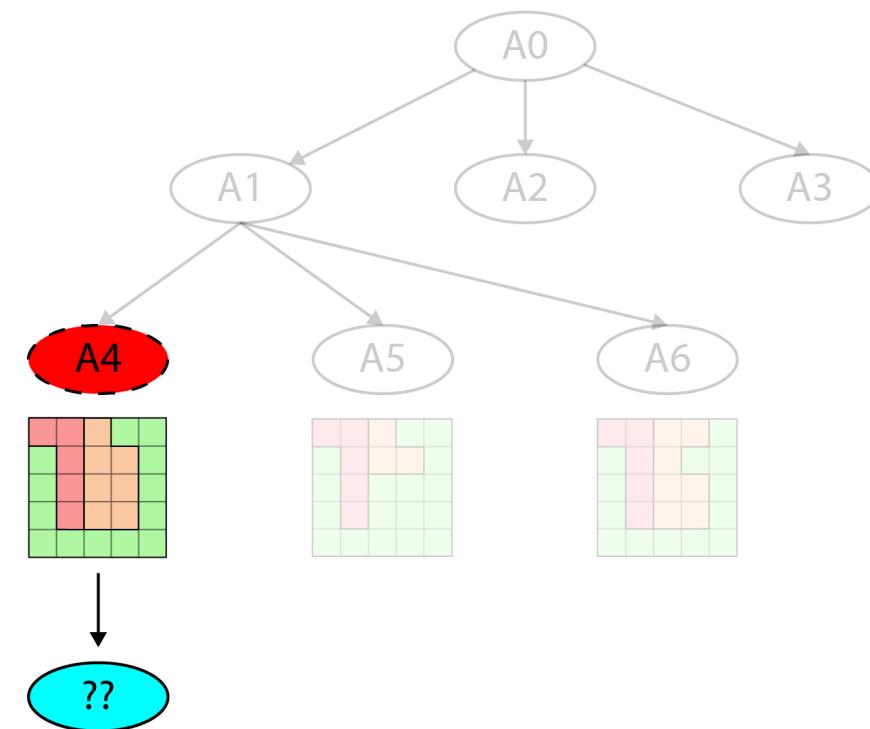
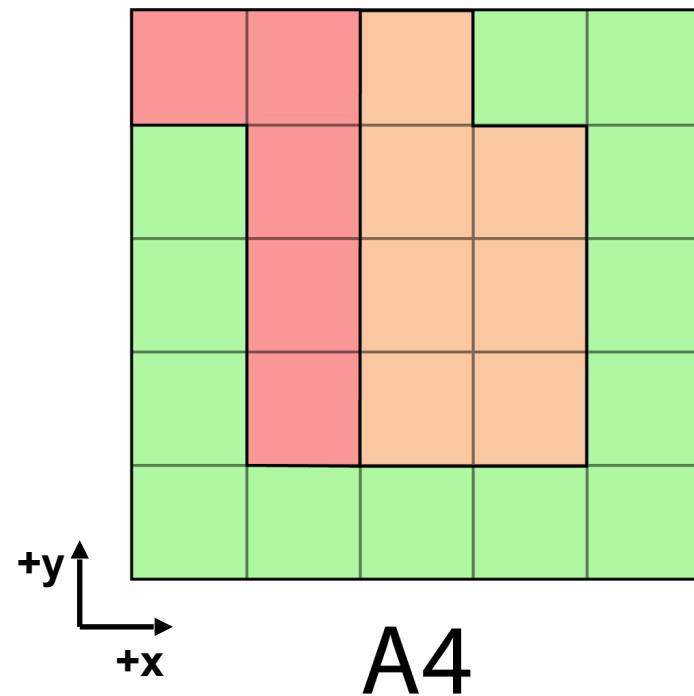
# Interactive Design Framework

Generate candidates of a 3-part interlocking assembly.



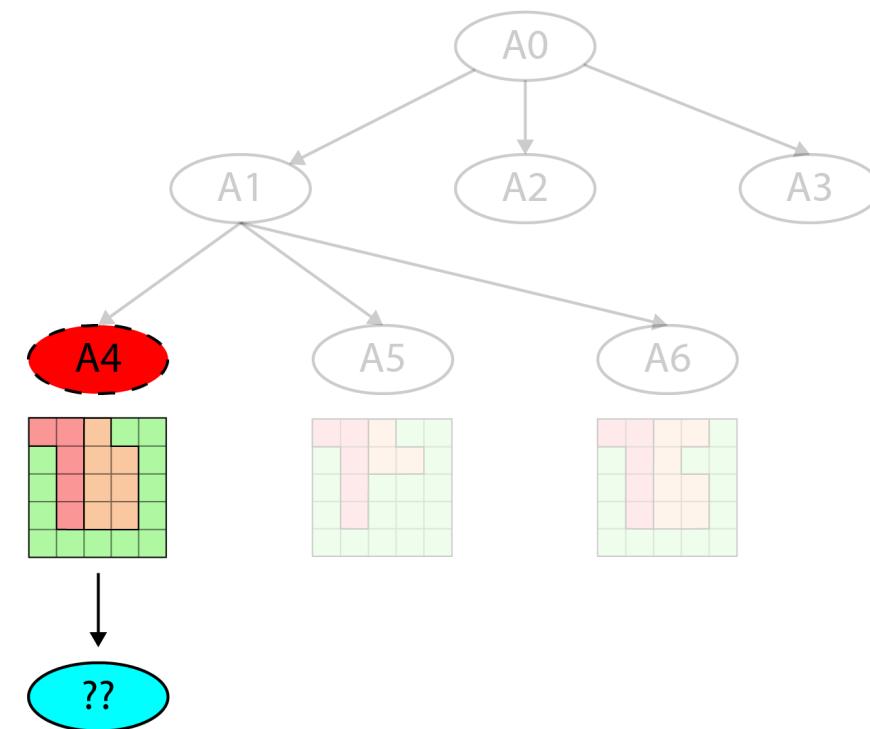
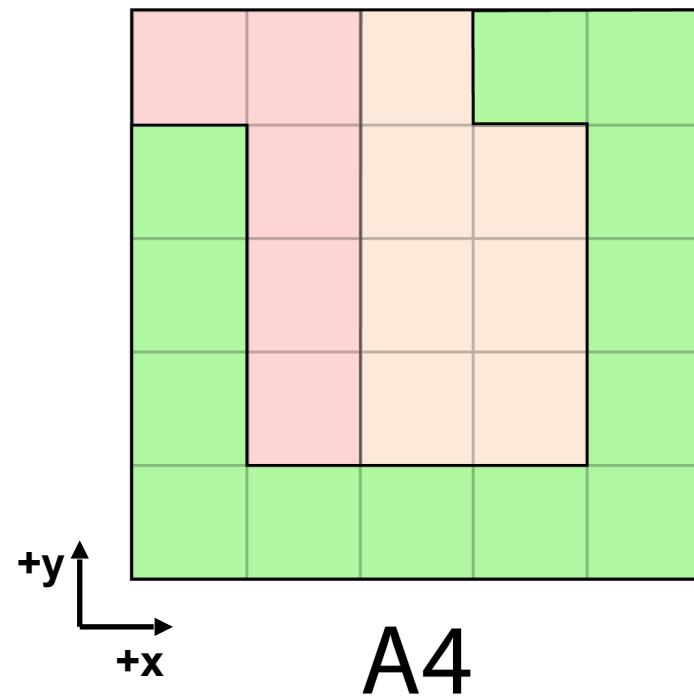
# Interactive Design Framework

Can we further generate a 4-part interlocking assembly?



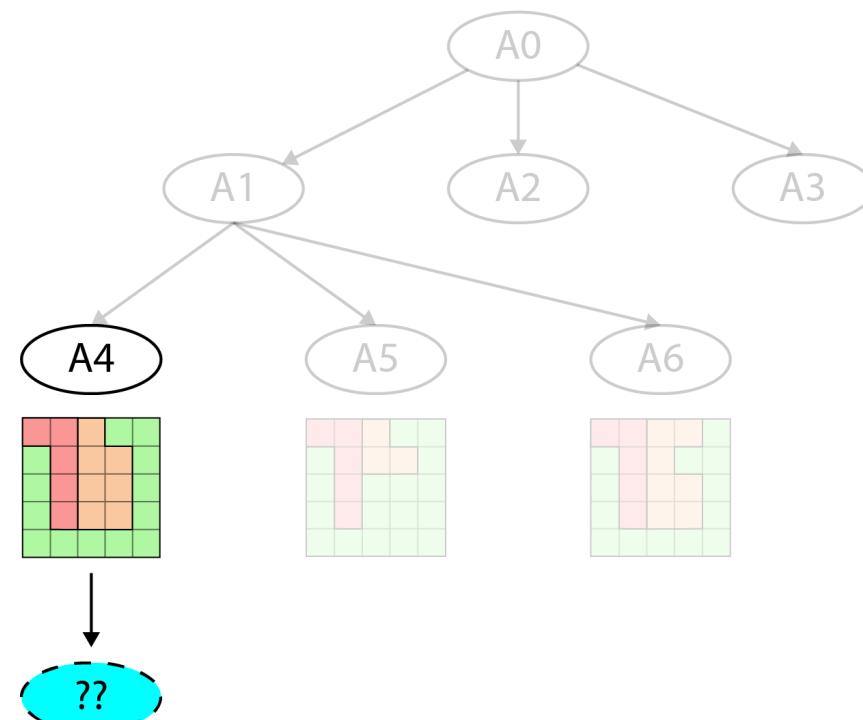
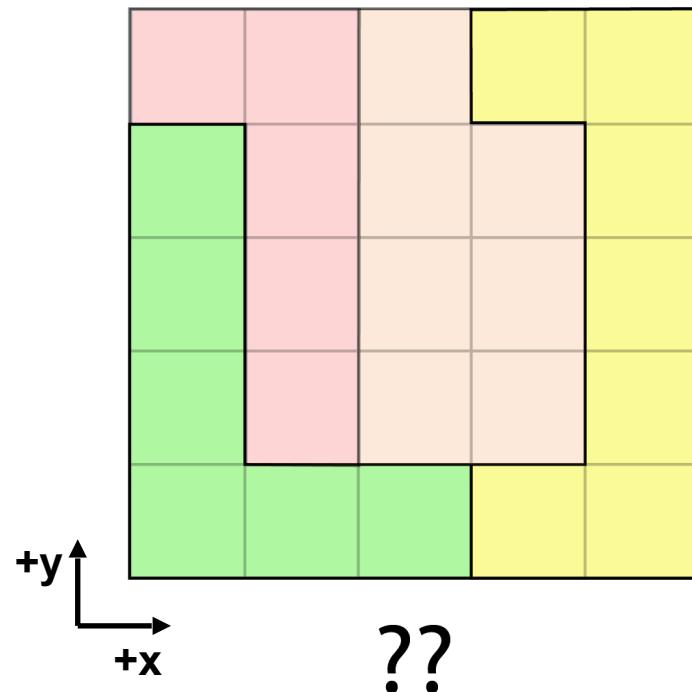
# Interactive Design Framework

Can we further generate a 4-part interlocking assembly?



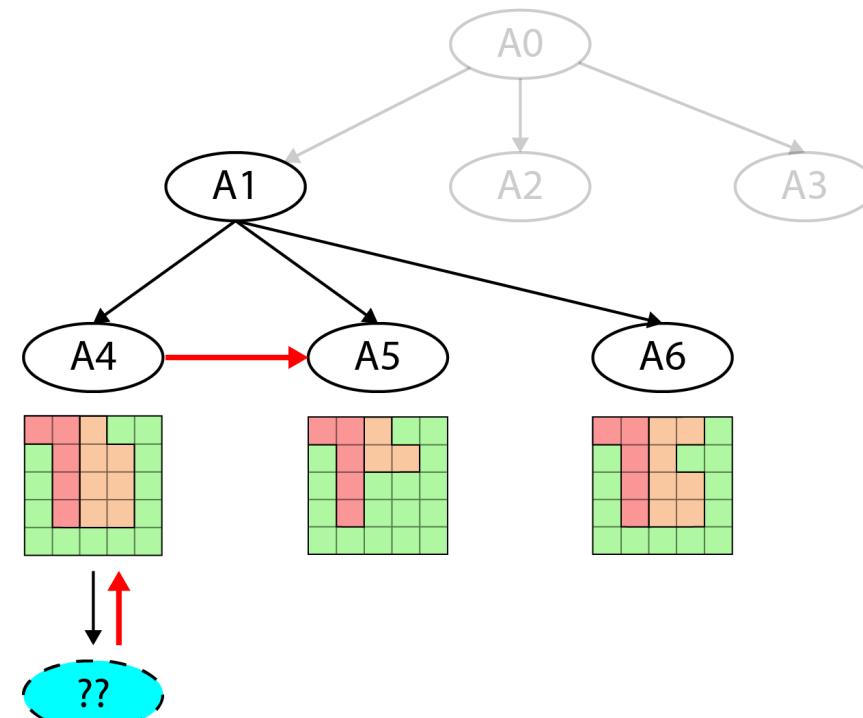
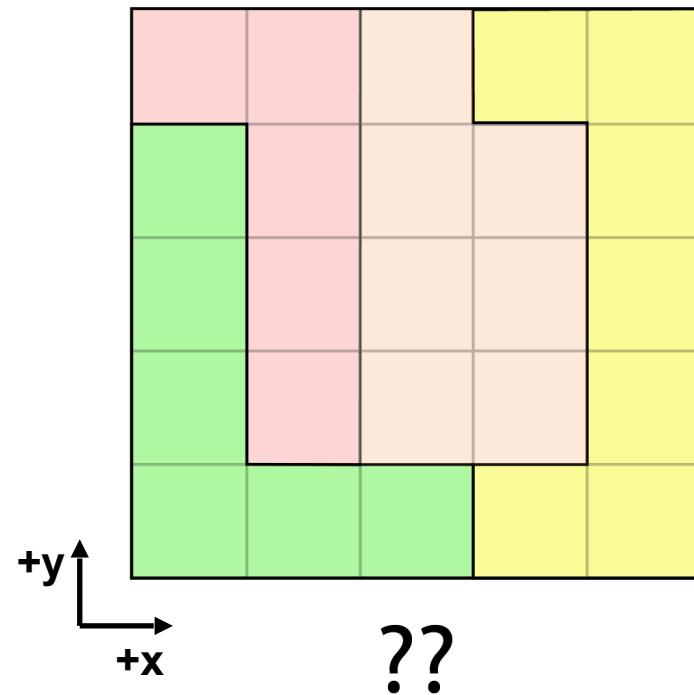
# Interactive Design Framework

No matter what the partition is, this candidate cannot be interlocking.



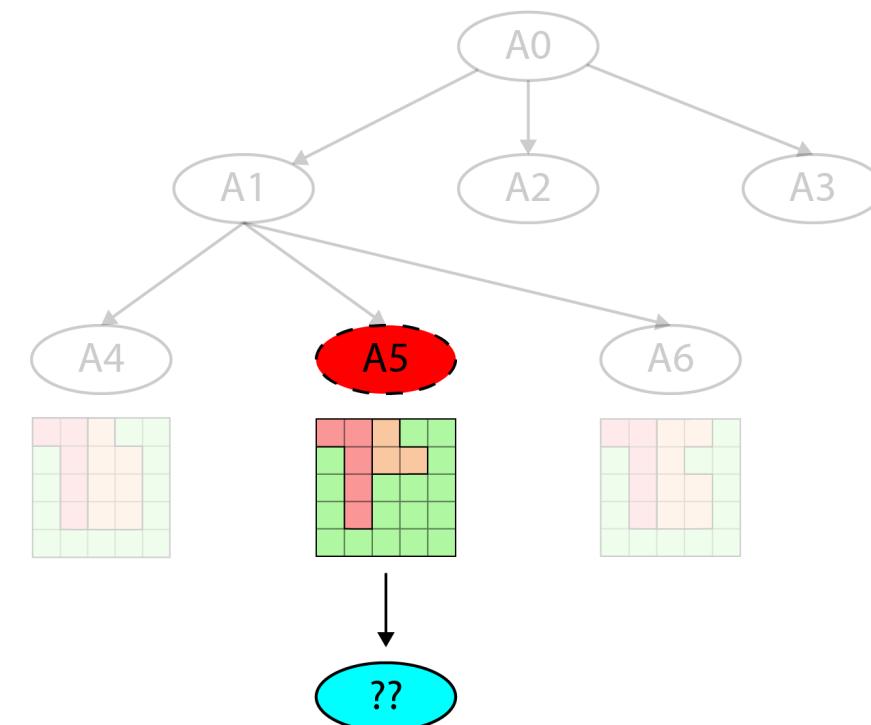
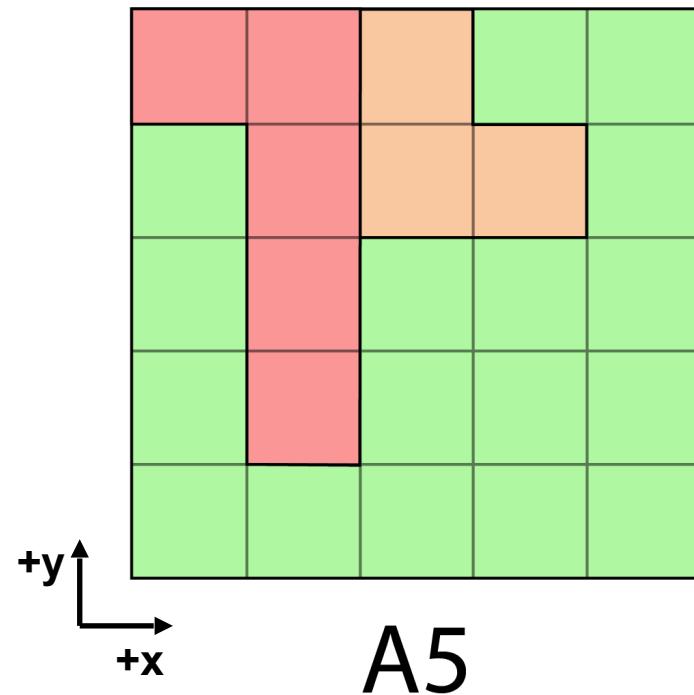
# Interactive Design Framework

So we need to do backtracking.

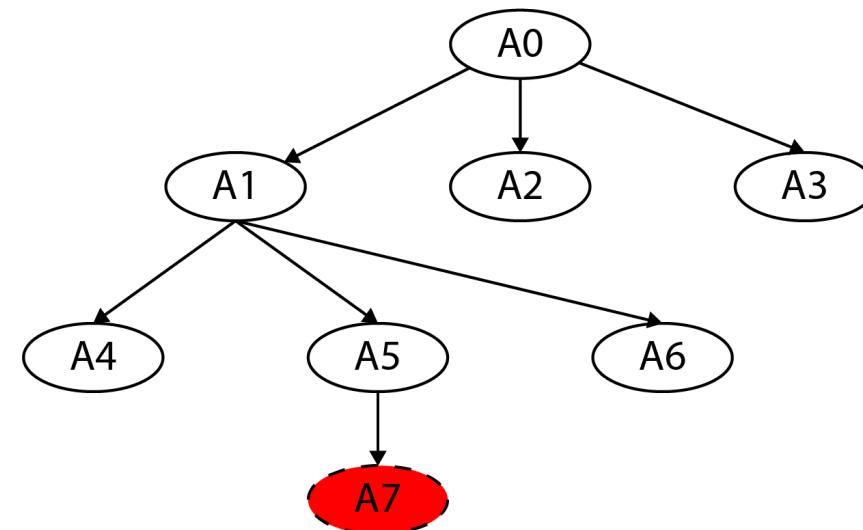
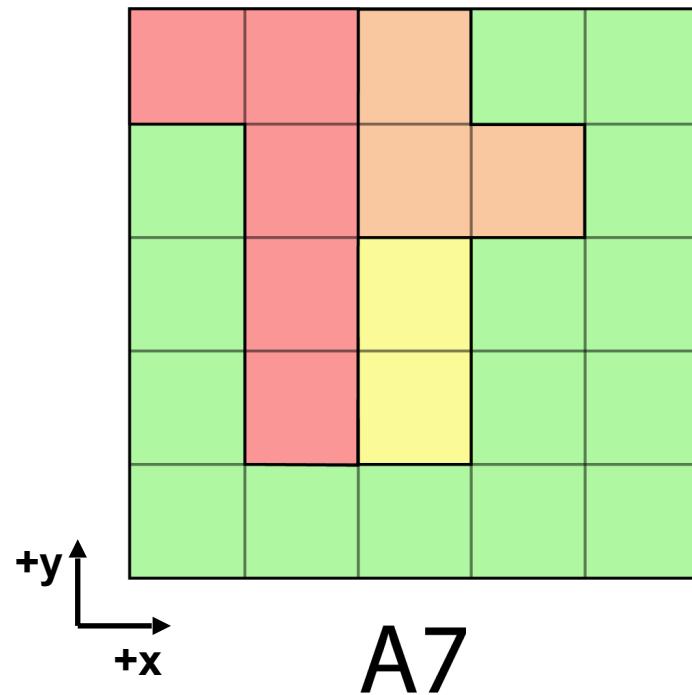


# Interactive Design Framework

Can we generate a 4-part interlocking assembly?

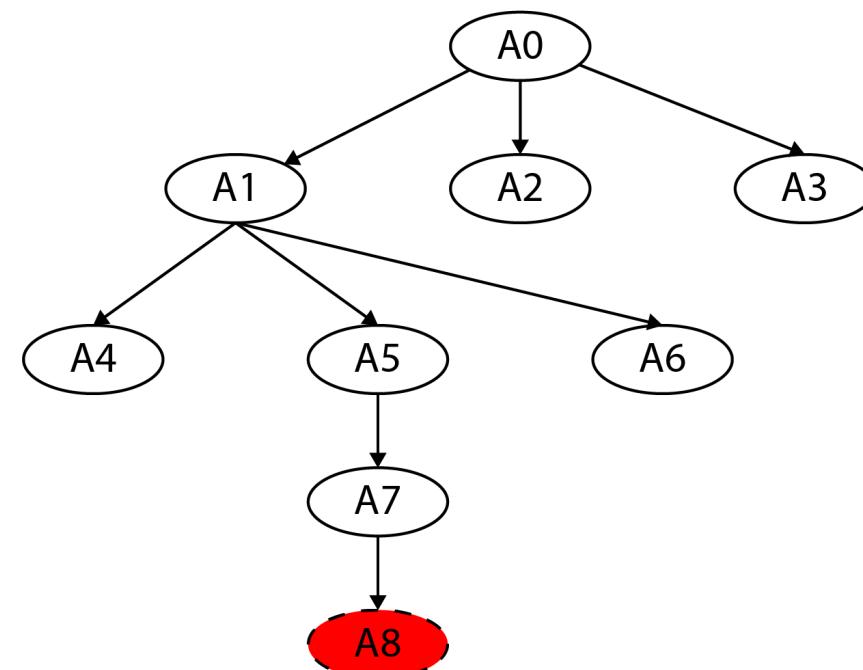
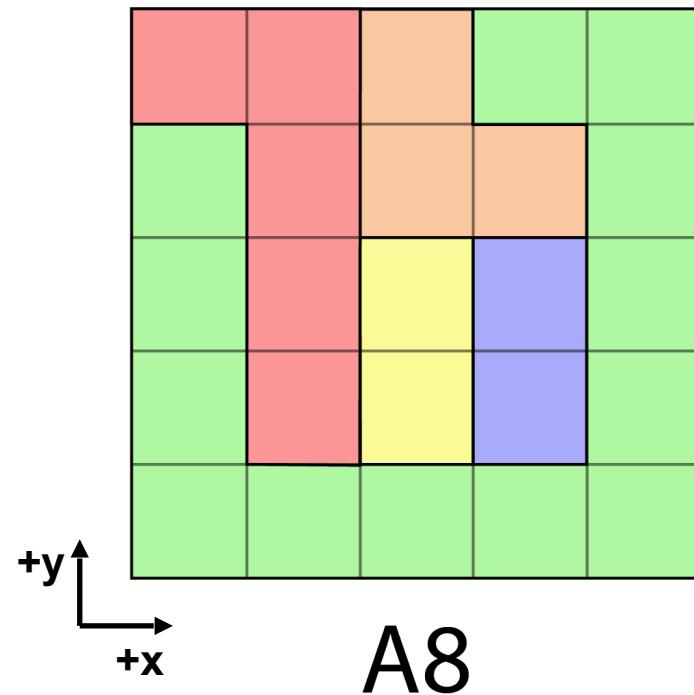


# Interactive Design Framework

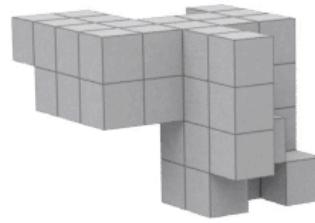


# Interactive Design Framework

Succeed to find a 5-part interlocking assembly and terminate.

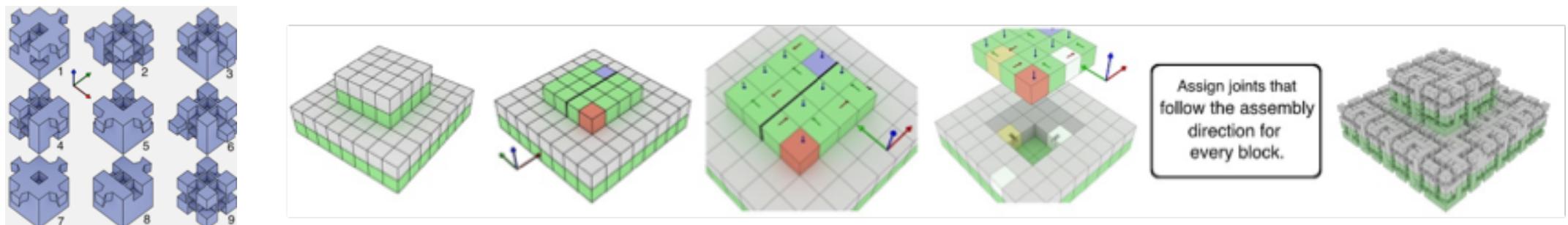


# Result



# Interlocking Voxels

- Zhang et al. designed a set of nine tileable blocks with integral joints, called interlocking voxels.



[Zhang et al. 2016]

# Conclusion

- A general workflow for stability optimization problems.
- Force and Kinematic-based Equilibrium method
- Lateral stability, scaffold-free assembly, globally interlocking assemblies.

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Thank you!