

a) Design of Spur Pair for the first stage :-

- In order to obtain all dimensions of the spur gear pinion and gear we must first find the module.
- Given Data :-

Given Material Al 7075, Factor of Safety = 2

$$S_{ut} = 570 \text{ N/mm}^2$$

(Z_{min})_{spur pinion} = 17, Z_g = 51, Pressur Angle = 20°

Solution :-

$$\text{Bending Stress} = \sigma_b = \frac{1}{3} S_{ut}$$

$$= \frac{1}{3} \times 570$$

$$= 211.6 \text{ N/mm}^2 \approx 210 \text{ N/mm}^2$$

Bending Strength can determined by the formula:-

$$S_b = m \cdot \sigma_b \cdot Y \cdot b$$

$$\sigma_b = 210 \text{ N/mm}^2$$

$$Y = \text{Lewis Form Factor} = \frac{0.484 + 2.87}{Z_p}$$

$$= \frac{0.484 + 2.87}{17}$$

$$= 0.6528$$

$$b = 10 \times m \quad (\text{Standard formula})$$

$$\therefore S_b = m \times 210 \times 0.6528 \times 10m$$

$$S_b = 1370.92 \text{ m}^2$$

We find the wear strength using the formula:-

$$S_w = b Q d_p K$$

$$b = 10m$$

$$Q = \frac{2 z_g}{z_g + 2 p}$$

$$= \frac{2 \times 51}{17 + 51}$$

$$Q = 1.5$$

$$K = \sigma_c^2 \sin \alpha \cos \alpha \left(\frac{1}{\epsilon_p} + \frac{1}{\epsilon_g} \right)$$

$$\sigma_c \text{ (For AL 7075)} = 280 \text{ MPa.}$$

$$\alpha = \text{Pressure angle} = 20^\circ$$

$$K = 280 \times \sin 20^\circ \times \cos 20^\circ \times (2.857 \times 10^{-5}) \\ = 0.514$$

$$d_p = m \times 2p$$

$$= m \times 17$$

$$= 17m.$$

$$\therefore S_w = 10m \times 1.5 \times 17m \times 0.514$$

$$S_w = 131.12 \text{ m}^2$$

* As Gear and Pinion are made of the same material, Pinion will be weak. Hence design must be done w.r.t pinion.

$$\text{Bending strength of pinion} = 1370.92 \text{ m}^2$$

$$\text{Wear strength of pinion} = 131.12 \text{ m}^2.$$

As $S_b > S_w$, failure will happen through wear.

Now, we calculate F_{eff} .

$$F_{eff} = \frac{K_a \cdot K_m \cdot F_t}{K_v}$$

$$F_t = \frac{P}{V}$$

$$P = 1130 \text{ W}$$

$$V = \frac{\pi d p n P}{60 \times 1000} \text{ m/s}$$

$$= \frac{\pi \times 17 \text{ m} \times 9330}{60 \times 1000}$$

$$V = 8.304 \text{ m/s.}$$

$$\therefore F_t = \frac{1130}{8.304 \text{ m}}$$

$$F_t = \frac{136.07}{\text{m}}$$

K_v = Velocity factor.

As for high speed application we consider value of K_v as
 $K_v = \frac{5.6}{5.6 + \sqrt{8.304 m}}$

$$K_v = \frac{5.6}{5.6 + \sqrt{8.304 m}}$$

$$F_{eff} = 1.2 \times 1.5 \times \frac{136.07}{m} \div \frac{5.6}{5.6 + \sqrt{8.304 m}}$$

$\left[\because K_a = 1.2, K_m = 1.5 \text{ based on table from VB Bhandari} \right]$

Given $f(s) = 2$.

Now, we create equation for finding module.

We know that :- $S_w = F_{eff} \times f(s)$.

$$113.12 m^2 = \left[1.2 \times 1.5 \times \frac{136.07}{m} \div \frac{5.6}{5.6 + \sqrt{8.304 m}} \right] \times 2$$

$$56.56 m^2 = \left[\frac{244.926}{m} \times \frac{5.6 + \sqrt{8.304 m}}{5.6} \right]$$

Now, we use bisection method for finding positive value of m that satisfies the given equation.

Using bisection method we find get $m = 1.6$

Based on following module, we obtain all other dimensions
of the spur pair.

- 1 Module = $m = 1.6 \text{ mm}$
- 2 $d_p = 27.2 \text{ mm} \approx 28 \text{ mm}$
- 3 $d_g = 82 \text{ mm}$
- 4 $b = 10 \times m = 10 \times 1.6 = 16 \text{ mm}$
- 5 Backlash = 0.24 mm
- 6 Centre to centre distance = $a = \frac{2p+2p}{2} = 55 \text{ mm}$
- 7 Addendum = $m = 1.6 \text{ mm}$
- 8 Dedendum = $1.25m = 2 \text{ mm}$
- 9 Clearance = $0.25m = 0.4 \text{ mm}$
- 10 Working depth = $2m = 3.2 \text{ mm}$
- 11 Whole depth = $2.25m = 3.6 \text{ mm}$
- 12 Tooth thickness = $1.5708m = 2.513 \text{ mm}$.