# Stability of Oscillatory Rotating Disk Boundary Layers

#### Scott Morgan

Supervisor: Dr. Christopher Davies

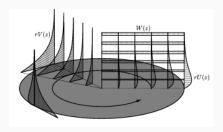
ERCOFTAC SIG33 Workshop 2017 21st June 2017



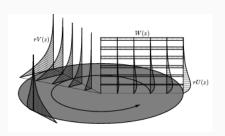
### **Structure**

- Part 1: What?
- Part 2: Why?
- **Part 3:** How?

# Stability of Oscillatory Rotating Disk Boundary Layers



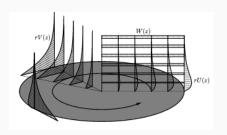
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$$\delta = \sqrt{\nu/\Omega_0}$$

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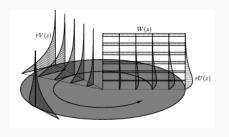


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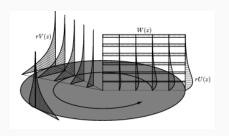
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  - Type II  $R_c \approx 440$

Malik (1986)

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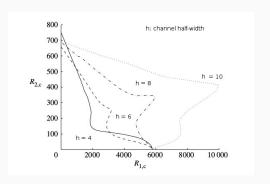
Malik (1986)

- Absolutely unstable for travelling disturbances:
  - $R_c \approx 507$

Lingwood (1995)

### Stability of Oscillatory Rotating Disk Boundary Layers

Adding oscillation to channel flow can be stabilising



Thomas et. al. (2011)

Stability of Oscillatory Rotating Disk Boundary Layers

Stability of Oscillatory Rotating Disk Boundary Layers

Dominant behaviour is Stokes layer for high-frequency, low amplitude oscillations.

# Why rotating disks?

 $\bullet \ \ \mathsf{Rotating} \ \mathsf{disks} \approx \mathsf{Swept} \ \mathsf{wings} \\$ 

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- ullet Rotating disks pprox Swept wings
  - Crossflow vortex suppression focus on stationary disturbances.

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- ullet Rotating disks pprox Swept wings
  - Crossflow vortex suppression focus on stationary disturbances.
- Other applications atmospheric, oceanic, chemical deposition, mixing

# How?

# Setup

• Three-dimensional base flow

$$\mathbf{U}_B=(U,V,W)$$

### Setup

• Three-dimensional base flow

$$\mathbf{U}_B = (U, V, W)$$

• Boundary conditions:

$$V(r, z = 0, t) = r\Omega(t)$$
  
=  $r(\Omega_0 + \epsilon\phi\cos(\phi t))$ 

 $\Omega_0$  - constant rotation rate  $\epsilon$  - angular displacement

 $\phi$  - oscillation frequency

• Retain steady scalings:

$$\delta^* = \sqrt{rac{
u^*}{\Omega_0^*}}, \qquad \quad au = rac{r_L^*\Omega_0^*t^*}{\delta^*}$$

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Similarity structure:

$$\mathbf{U}_B = \left(\frac{U^*}{r^*\Omega_0^*}, \frac{V^*}{r^*\Omega_0^*}, \frac{W^*}{\delta^*\Omega_0^*}\right)$$

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Three parameters:

$$(R_k, R_s, \varphi)$$

(Rs o 0 recovers steady case)

• Three approaches:

- Three approaches:
  - 1. Floquet Theory

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- Solve Navier-Stokes equations using velocity-vorticity formulation.

 $1. \ \, \textbf{Floquet Theory}$ 

# Floquet Theory

• Normal mode approximation:

$$u(r,\theta,z, au)\sim \hat{u}(z, au)e^{ilpha r}e^{\mu au}e^{in heta}$$

•  $\hat{u}(z,\tau)$  periodic

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$$\sum_{k=-K}^{K} \mathcal{L}_{k}\{\mu,\alpha;\textbf{n},\textbf{R}_{k},\textbf{R}_{s},\varphi\} e^{ik\tau} = 0$$

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 $\bullet$  Specify  $\mu$  or  $\alpha$  as real and solve for the other.

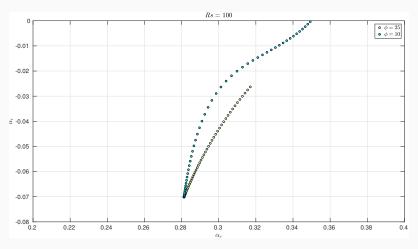
## **Floquet Results**

Steady case: 
$$R_k = 500$$
,  $n = 32$ ,  $\alpha_i \approx -0.07$ 

# Floquet Results

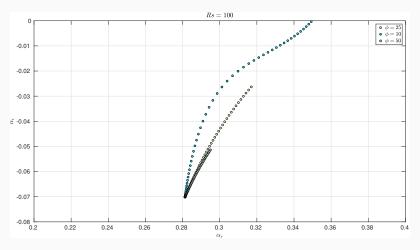
$$R_{\it k}=$$
 500,  $\it n=$  32,  $\it arphi=$  25,  $\it R_{\it s}\in [0,100]$ 

# Floquet Results



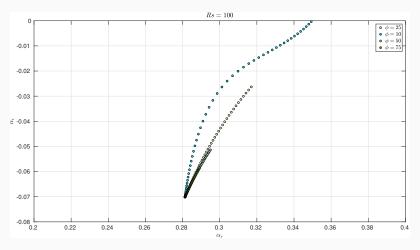
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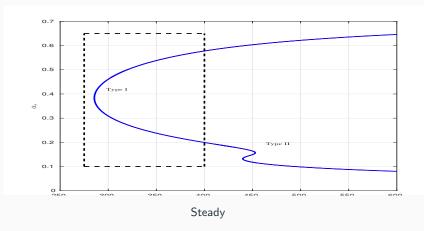


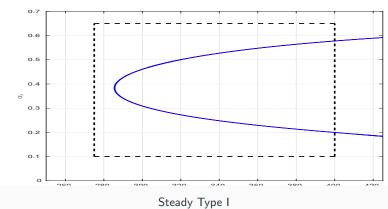
$$\textit{R}_{\textit{k}} = 500$$
 ,  $\textit{n} = 32$  ,  $\textit{\varphi} \in \{10, 25, 50\}$  ,  $\textit{R}_{\textit{s}} \in [0, 100]$ 

# Floquet Results

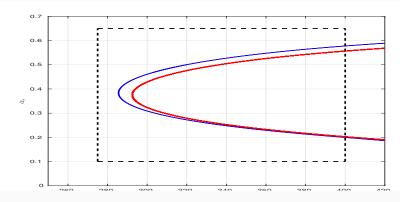


$$R_k = 500, \ n = 32, \ \varphi \in \{10, 25, 50, 75\}, \ R_s \in [0, 100]$$

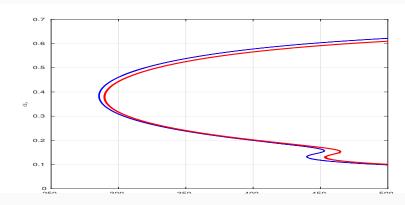




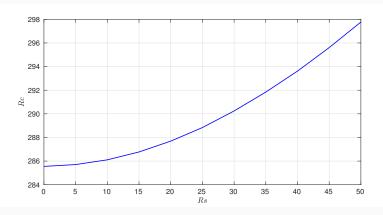
Steady Type



Type I:  $\mathit{Rs} = 20$ ,  $\varphi = 25$ 



Full curve:  $\mathit{Rs} = 20$ ,  $\varphi = 25$ 



Critical Type I  $R_c$  for arphi=25 and  $Rs\in[0,50]$ 

# Floquet Results

• Clear reduction in spatial growth rates for small Rs.

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- Larger Rs being explored currently preliminary results indicate intricate behaviour in neutral curves when  $U_w > 1$  if

$$G(0, \tau) = 1 + U_w \cos\left(\frac{\varphi}{R_k}\right)$$

# **Approaches**

2. Linear DNS

• Measure the response of the flow to a *stationary* disturbance.

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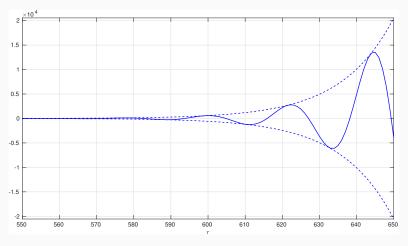
•  $\theta$  coordinate changes:

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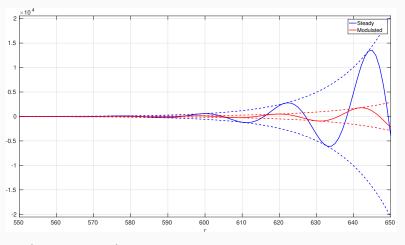
$$\zeta(r,\theta_0, au) = e^{\lambda r^2} e^{in\theta_0} e^{in\int^{ au} \Omega(\tilde{ au})}$$

• Measure the response of the flow to a *stationary* disturbance.



$$\textit{u(r,z}=0,\tau > \textit{T}_{\textit{c}})$$
 - steady:  $\textit{R}_{\textit{k}} = 500, \, \textit{n} = 32$ 

• Measure the response of the flow to a stationary disturbance.



$$u(r,z=0, au>T_c)$$
 - modulated:  $R_k=500$ ,  $n=32$ ,  $Rs=50$ ,  $arphi=25$ 

### **DNS**

• Normal mode approximation:

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•  $-\alpha_i$  gives radial growth rate.

# DNS vs. Floquet

#### DNS

• Normal mode approximation:

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#### **Floquet**

• Normal mode approximation:

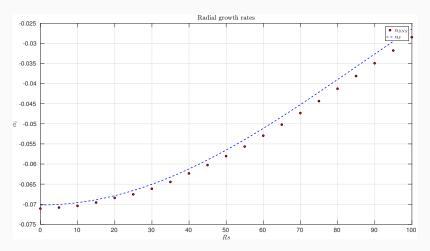
$$u(r,z,\tau)\sim \hat{u}(z,\tau)e^{i\alpha r}e^{\mu\tau}$$

 Harmonic decomposition gives eigenvalue problem:

$$\sum_{k=-K}^{K} \mathcal{L}_{k}\{\mu,\alpha\} e^{ik\tau} = 0$$

• Specify  $\mu$  or  $\alpha$  as real and solve for the other.

# DNS vs. Floquet



Radial Growth Rates:  $R_k = 500$ , n = 32,  $\varphi = 25$ ,  $Rs \in \{0, 100\}$ 

### **Current & Future Work**

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- Look at parallels between oscillation and surface roughness.
- Experimental confirmation.
- Explore torsional oscillations.

# Thank You

# **Approaches**

3 Frozen Flow Analysis

ullet Freeze flow, treat au as parameter:

$$p(r,\theta,z,\tau) = \hat{p}(z;\tau)e^{i\alpha r}e^{in\theta}e^{-i\int^{\tau}\omega}$$

where  $\hat{\phi}$  is slowly varying.

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• Dispersion relation:

$$\mathcal{D}(\alpha,\omega;n,R_k,R_s,\varphi,\tau)=0 \tag{1}$$

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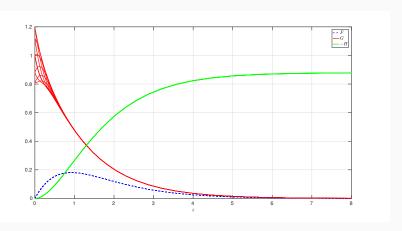
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• 
$$\frac{1}{T} \int_0^T \alpha(\tau) \approx \alpha_F$$

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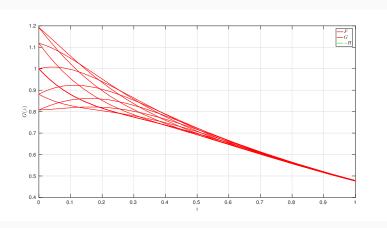
# **Typical Mean Flow Variation**



Base flow variation for  $\mathit{Rs}=10$ ,  $\varphi=50$ 

ullet Zero average deviation from steady state across a period.  $\int_0^T {f U} = 0$ 

# **Typical Mean Flow Variation**



Azimuthal variation near wall for  $\mathit{Rs} = 10$ ,  $\varphi = 50$ 

ullet Zero average deviation from steady state across a period.  $\int_0^T \mathbf{U} = 0$ 

# Connections with the Stokes Layer

Write

$$\mathbf{U}^{T} = \mathbf{U}^{S} + \mathbf{U}^{M}$$
$$= \mathbf{U}^{S} + \epsilon \mathbf{U}_{1} + \mathcal{O}(\epsilon^{2})$$

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$$\delta_{s}=\sqrt{rac{1}{arphi}}$$

Rescale

$$\tilde{z} = \frac{z}{\delta_s}, \quad \tilde{\tau} = \frac{\varphi}{R}\tau$$

• Near the wall we have

$$\left(\frac{1}{\delta^{2}}\right)\frac{\partial F}{\partial \tilde{\tau}} = \left(\frac{1}{\delta^{2}}\right)F'' + \mathcal{O}\left(\delta^{-1}\right)$$
$$\left(\frac{1}{R\delta^{2}}\right)\frac{\partial G}{\partial \tilde{\tau}} = \left(\frac{1}{\delta^{2}}\right)G'' + \mathcal{O}\left(\delta^{-1}\right)$$
$$H \sim \delta F$$

with

$$F(0, \tilde{\tau}) = H(0, \tilde{\tau}) = 0, \quad G(0, \tilde{\tau}) = \cos(\tilde{\tau})$$
  
 $F \to 0 \quad G \to 0 \quad \text{as} \quad \tilde{z} \to \infty$ 

To dominant balance:

$$\begin{split} \frac{\partial F}{\partial \tilde{\tau}} &= F'', \quad \text{ with } \quad F(0) = F(\tilde{z} \to \infty) = 0 \\ \frac{1}{R} \frac{\partial G}{\partial \tilde{\tau}} &= G'', \quad \text{ with } \quad G(0) = \cos(\tilde{\tau}), \quad G(\tilde{z} \to \infty) = 0 \end{split}$$

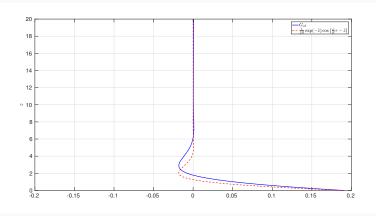
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• Which gives Stokes layer:

$$F = 0$$

$$G = \frac{1}{R} \exp(-\tilde{z}) \cos(\tilde{\tau} - \tilde{z})$$



Comparison between Stokes layer profile and base flow variation

#### **DNS** - Wall Motion

• Wall displacement for stationary forcing (steady, rotating frame):

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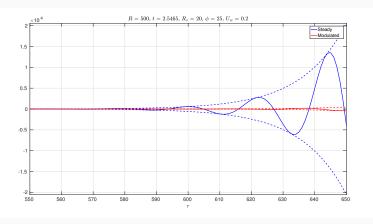
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• Forcing stationary with respect to modulated disk:

$$\zeta(r,\theta_0,\tau) = e^{\lambda r^2} e^{in \int^{\tau} \Omega(\tilde{\tau})} e^{in\theta_0}$$

• Prescribe wall motion  $\zeta(r,\theta_0,\overline{\tau}) = e^{\lambda r^2} e^{in \int^{\tau} \Omega(\tilde{\tau})} e^{in\theta_0}$ 

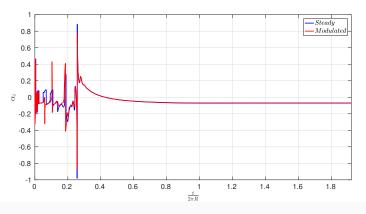
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Radial evolution: R = 500, n = 32, Rs = 20,  $\varphi = 25$ 

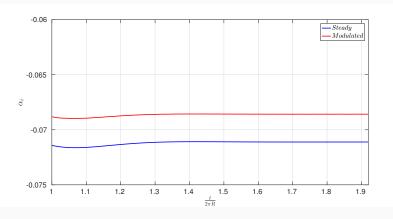
Receptivity issues.

• Calculate  $\alpha = -\frac{i}{A} \frac{\partial A}{\partial t}$  at fixed r.



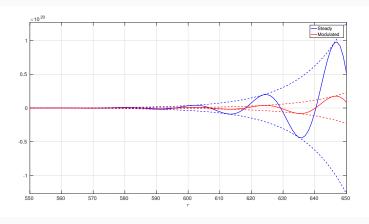
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• Calculate  $\alpha = -\frac{i}{A} \frac{\partial A}{\partial t}$  at fixed r.



$$R = 500$$
,  $n = 32$ ,  $Rs = 20$ ,  $\varphi = 25$ 

• Exponential growth reconstructed from  $e^{i\alpha r}$ 



$$R=500$$
,  $n=32$ ,  $Rs=20$ ,  $arphi=25$ 

#### **DNS** - Results

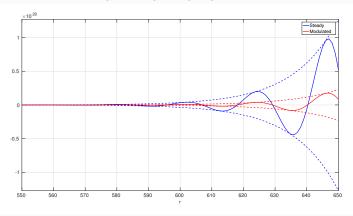
• Exactly prescribe  $(\alpha, \mu, \psi)$  from Floquet theory at inflow.

$$\psi(r,\theta,z,\tau) = \hat{\psi}(z,\tau)e^{\mu\tau}e^{i\alpha r}e^{in\theta}$$

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Radial evolution: R = 500, n = 32, Rs = 20,  $\varphi = 25$ 

#### **DNS** - Results

• Calculate  $\alpha = -\frac{i}{A} \frac{\partial A}{\partial t}$  for fixed r.

Stationary Forcing		
Rs	$\varphi$	α
0	N/A	0.2821 - 0.0712i
10	$\varphi = 25$	0.2820 - 0.0701i
	$\varphi=50$	0.2820 - 0.0709i
20	$\varphi = 25$	0.2821 - 0.0686i
	arphi=50	0.2817 - 0.0702i

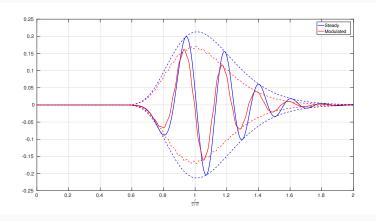
Inflow Prescribed Forcing		
Rs	$\varphi$	α
0	N/A	0.2821 - 0.0712i
10	$\varphi = 25$	0.2820 - 0.0701i
	$\varphi = 50$	0.2820 - 0.0709i
20	$\varphi = 25$	0.2821 - 0.0685i
	$\varphi = 50$	0.2818 - 0.0702i

# **DNS** - Impulsive Forcing

 $\bullet$  Prescribe impulsive wall motion  $\zeta(r,\theta_0,\tau)=e^{\lambda r^2}e^{-\sigma t^2}$ 

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Temporal evolution: R=350, n=32, Rs=20,  $\varphi=25$