

ME 596 Spacecraft Attitude Dynamics and Control

Review of Particle Dynamics

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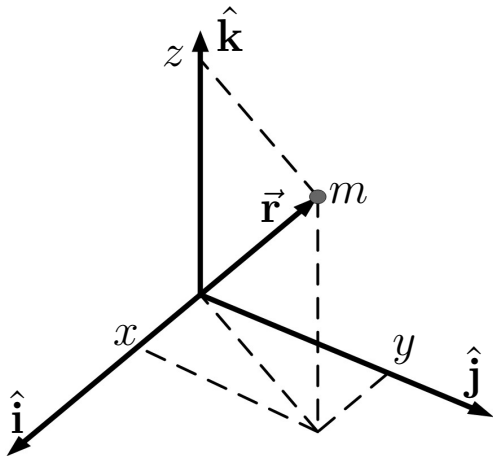
October 18, 2021

$$\vec{\mathbf{f}} = m\vec{\mathbf{a}}$$

- ▶ Applicable to mass particles with *constant* mass and *zero* volume, and is extensible to finite-size, constant-mass rigid bodies
- ▶ Not as simple as it looks
 - ▶ Determination of the force usually requires a free-body diagram, which can be extremely complicated
 - ▶ Determination of the acceleration sometimes involves complicated kinematics
 - ▶ Acceleration is second derivative of position vector with respect to time
 - ▶ Position vector is sometimes expressed in a non-inertial reference frame

Reference Frame and Position Vector

$$\vec{r} = x\hat{i} + y\hat{j} + z\hat{k}$$



Position vector must be from inertial origin to mass particle

If expressed in terms of inertial frame components, then differentiation is easy

An inertial origin is a point that is not accelerating with respect to any other inertial origin

- ▶ Alternatively, an inertial origin is a point for which Newton's laws are applicable
- ▶ There is no known inertial origin, but for most problems an origin can be found that is “inertial enough”
- ▶ For some problems, an Earth-fixed reference point is inertial enough, whereas for others, the rotation of the Earth must be taken into account

Inertial Reference Frame

An inertial reference frame is a set of three unit vectors that are mutually perpendicular, with their origins at a single inertial origin, whose directions remain fixed with respect to inertial space

- ▶ Alternatively, an inertial reference frame is a frame for which Newton's laws are applicable
- ▶ Usually the dynamics and control analyst must determine the simplest frame that is “inertial enough”
- ▶ What reference frames would be “inertial enough” for these problems?
 - Paper airplane, Cessna, B2, container ship, Falcon Heavy, Voyager, Rama, Oumuamua

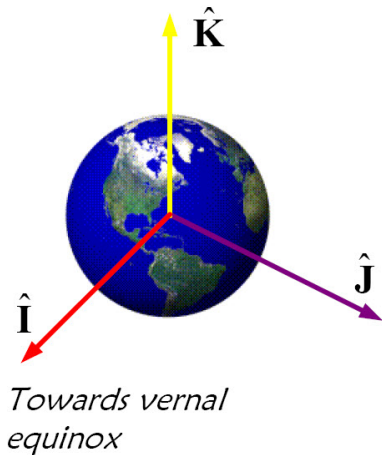
- ▶ A reference frame is a set of three mutually perpendicular (orthogonal) unit vectors
- ▶ Typical notations include

$$\hat{\mathbf{i}} \hat{\mathbf{j}} \hat{\mathbf{k}}, \quad \hat{\mathbf{I}} \hat{\mathbf{J}} \hat{\mathbf{K}}, \quad \hat{\mathbf{e}}_1 \hat{\mathbf{e}}_2 \hat{\mathbf{e}}_3, \quad \hat{\mathbf{b}}_1 \hat{\mathbf{b}}_2 \hat{\mathbf{b}}_3, \quad \hat{\mathbf{o}}_1 \hat{\mathbf{o}}_2 \hat{\mathbf{o}}_3$$

- ▶ Typical reference frames of interest for vehicles include
 - ECI (Earth-centered inertial)
 - Perifocal (Earth-centered, orbit-based inertial)
 - ECEF (Earth-centered, Earth-fixed, rotating)
 - Orbital (Earth-centered, orbit-based, rotating)
 - Body (spacecraft-fixed, rotating)

Earth-Centered Inertial Frame (ECI)

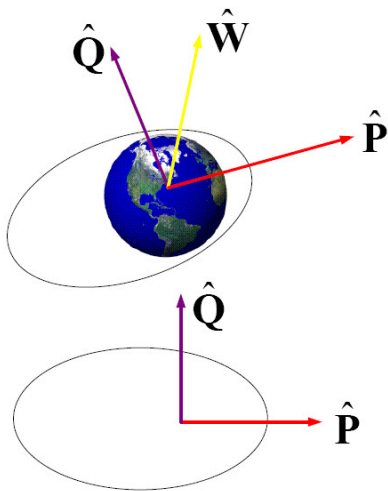
- ▶ The $\hat{\mathbf{I}}$ -axis is in the vernal equinox direction (i.e., points to the Sun on the first day of Spring)
- ▶ The $\hat{\mathbf{K}}$ -axis is Earth's rotation axis, perpendicular to the equatorial plane
- ▶ The $\hat{\mathbf{J}}$ -axis is in the equatorial plane and completes the "triad" of mutually perpendicular unit vectors



Note that $\hat{\mathbf{I}} \times \hat{\mathbf{J}} = \hat{\mathbf{K}}$ by the "right-hand rule"

Perifocal Frame

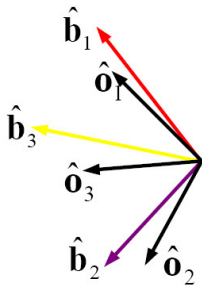
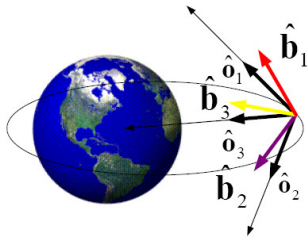
- ▶ Earth-centered, orbit-based, inertial
- ▶ The $\hat{\mathbf{P}}$ -axis is in the periapsis direction
- ▶ The $\hat{\mathbf{W}}$ -axis is perpendicular to the orbital plane (i.e., direction of orbit angular momentum vector, $\vec{r} \times \vec{v}$)
- ▶ The $\hat{\mathbf{Q}}$ -axis is in the orbital plane and completes the “triad” of mutually perpendicular unit vectors



Note that $\hat{\mathbf{P}} \times \hat{\mathbf{Q}} = \hat{\mathbf{W}}$ by the “right-hand rule”

Orbital Frame

- ▶ Similar to “roll-pitch-yaw” frame, for spacecraft
- ▶ Spacecraft-centered, orbit-based, rotating (non-inertial)
- ▶ The \hat{o}_3 -axis is in the nadir direction (i.e., direction of Earth, $-\vec{r}$)
- ▶ The \hat{o}_2 -axis is in the negative orbital normal direction (i.e., in the $-\hat{W}$ direction)
- ▶ The \hat{o}_1 -axis completes the “triad” of mutually perpendicular unit vectors, and is in the velocity vector direction for circular orbits

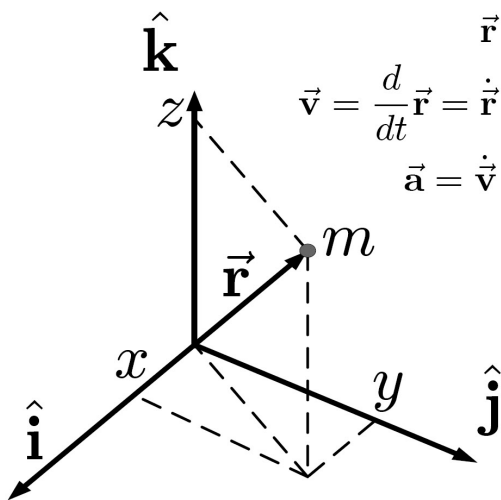


Note that $\hat{o}_1 \times \hat{o}_2 = \hat{o}_3$ by the “right-hand rule”

Vector, Frame, and Matrix Notation (preview)

Symbol	Meaning
\vec{v}	vector, an abstract mathematical object with direction and length
$\{\hat{\mathbf{i}}_1, \hat{\mathbf{i}}_2, \hat{\mathbf{i}}_3\}$	the three unit base vectors of a reference frame
\mathcal{F}_i	the reference frame with base vectors $\{\hat{\mathbf{i}}_1, \hat{\mathbf{i}}_2, \hat{\mathbf{i}}_3\}$
$\{\hat{\mathbf{i}}\}$	a column matrix whose 3 elements are the unit vectors of \mathcal{F}_i
\mathbf{v}_i	a column matrix whose 3 elements are the components of the vector \vec{v} expressed in \mathcal{F}_i
\mathbf{v}_b	a column matrix whose 3 elements are the components of the vector \vec{v} expressed in \mathcal{F}_b
\mathbf{R}^{bi}	rotation matrix that transforms vectors from \mathcal{F}_i to \mathcal{F}_b
θ	a column matrix whose 3 elements are the Euler angles $\theta_1, \theta_2, \theta_3$
$\vec{\omega}$	an angular velocity vector
$\vec{\omega}^{bi}$	angular velocity of \mathcal{F}_b with respect to \mathcal{F}_i
ω_b^{bi}	angular velocity of \mathcal{F}_b with respect to \mathcal{F}_i expressed in \mathcal{F}_b
ω_a^{bi}	angular velocity of \mathcal{F}_b with respect to \mathcal{F}_i expressed in \mathcal{F}_a

Position, Velocity, and Acceleration Vectors



$$\vec{r} = x\hat{i} + y\hat{j} + z\hat{k}$$

$$\vec{v} = \frac{d}{dt}\vec{r} = \dot{\vec{r}} = \dot{x}\hat{i} + \dot{y}\hat{j} + \dot{z}\hat{k}$$

$$\vec{a} = \dot{\vec{v}} = \ddot{x}\hat{i} + \ddot{y}\hat{j} + \ddot{z}\hat{k}$$

Derivatives are simple because unit vectors are constant, in direction and magnitude

Application of $\vec{f} = m\vec{a}$

Need to know the components of the force vector

$$\begin{aligned}\vec{f} &= f_x \hat{\mathbf{i}} + f_y \hat{\mathbf{j}} + f_z \hat{\mathbf{k}} \\ &= m \left(\ddot{x} \hat{\mathbf{i}} + \ddot{y} \hat{\mathbf{j}} + \ddot{z} \hat{\mathbf{k}} \right)\end{aligned}$$

\Rightarrow

$$f_x = m\ddot{x}$$

$$f_y = m\ddot{y}$$

$$f_z = m\ddot{z}$$

Three second-order ordinary differential equations.

Linearity and coupling depend on nature of forces.

Suppose the forces take the following form

$$f_x = -k_{px}x - k_{dx}\dot{x}$$

$$f_y = -k_{py}y - k_{dy}\dot{y}$$

$$f_z = -k_{pz}z - k_{dz}\dot{z}$$

Here the “ k ” terms are feedback gains, whose values are selected by the d&c analyst.

The k_p terms are *proportional* to the position errors, and the k_d terms are proportional to the velocity errors, or to the *derivatives* of the position errors, hence the controller is a “PD” controller

Equations of Motion with PD Control

First, define some new variables, termed the *states*

$$\begin{array}{ll} x_1 = x & \Rightarrow \dot{x}_1 = x_4 \\ x_2 = y & \dot{x}_2 = x_5 \\ x_3 = z & \dot{x}_3 = x_6 \\ x_4 = \dot{x} & \dot{x}_4 = f_x/m = \frac{1}{m} (-k_{px}x_1 - k_{dx}x_4) \\ x_5 = \dot{y} & \dot{x}_5 = f_y/m = \frac{1}{m} (-k_{py}x_2 - k_{dy}x_5) \\ x_6 = \dot{z} & \dot{x}_6 = f_z/m = \frac{1}{m} (-k_{pz}x_3 - k_{dz}x_6) \\ \downarrow & \\ \mathbf{x} \in \mathbf{R}^6 & \dot{\mathbf{x}} = \mathbf{f}(\mathbf{x}) \\ \boxed{\mathbf{x} \text{ is the state vector}} & \dot{\mathbf{x}} = \mathbf{f}(\mathbf{x}; \mathbf{x}_0, \mathbf{k}) \end{array}$$

Because of the unique form of these equations, they can be written in matrix form

- Each “right-hand side” appears as a summation of constants multiplying states, with the states always appearing linearly
- Thus, the equations comprise a system of linear, constant-coefficient, ordinary differential equations

$$\dot{\mathbf{x}} = \mathbf{f}(\mathbf{x}) = \mathbf{Ax}$$

Linear System (PD) (continued)

Simple rearrangement of the equations of motion leads to the block matrix form:

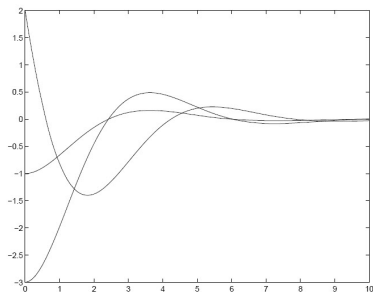
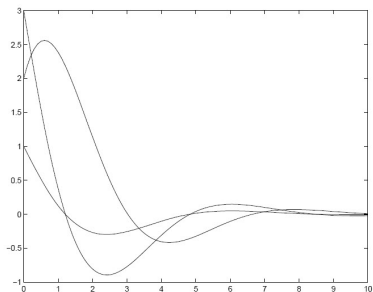
$$\begin{aligned}\dot{\mathbf{x}} &= \left[\begin{array}{ccc|ccc} 0 & 0 & 0 & 1 & 0 & 0 \\ 0 & 0 & 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 0 & 0 & 1 \\ \hline -k_{px}/m & 0 & 0 & -k_{dx}/m & 0 & 0 \\ 0 & -k_{py}/m & 0 & 0 & -k_{dy}/m & 0 \\ 0 & 0 & -k_{pz}/m & 0 & 0 & -k_{dz}/m \end{array} \right] \mathbf{x} \\ &= \left[\begin{array}{cc} \mathbf{0} & \mathbf{1} \\ -\text{diag}(\mathbf{k}_p)/m & -\text{diag}(\mathbf{k}_d)/m \end{array} \right] \mathbf{x} \\ &= \mathbf{A}\mathbf{x}\end{aligned}$$

Because \mathbf{A} is constant, the system is easily solved:

$$\mathbf{x}(t) = e^{\mathbf{A}t}\mathbf{x}_0$$

Linear System (PD) (continued)

The nature of the PD controller is that it causes all of the states to approach zero asymptotically; thus it is a stable controller

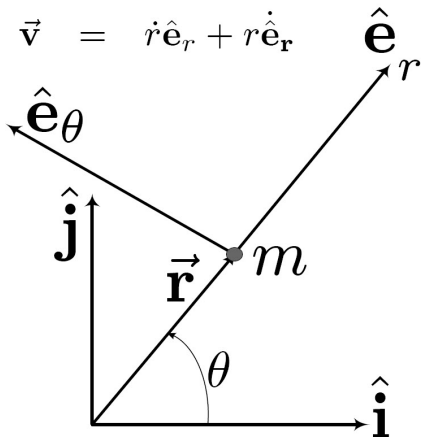


Note poor quality of these graphs. Not acceptable for technical presentations. How to improve?

Polar Coordinates, Rotating Reference Frame

$$\vec{r} = r\hat{e}_r$$

$$\vec{v} = \dot{r}\hat{e}_r + r\dot{\hat{e}}_r$$



Convenient to visualize frame origin at particle, but position vector must be from inertial origin.

Unit vectors of rotating reference frame are constant in length, *but not in direction*

*Unit vectors **are** orthogonal, in spite of distortion that appears in figure!*

Rate of Change of the Unit Vectors

- The unit vectors change because the angle θ changes
- Denote small change in time t and corresponding change in angle θ with Δt and $\Delta\theta$
- Consequently, the various derivatives can be written as

$$\dot{\hat{\mathbf{e}}}_r = \lim_{\Delta t \rightarrow 0} \frac{\Delta \hat{\mathbf{e}}_r}{\Delta t} = \lim_{\Delta t \rightarrow 0} \frac{\Delta\theta}{\Delta t} \hat{\mathbf{e}}_\theta = \dot{\theta} \hat{\mathbf{e}}_\theta$$

$$\dot{\hat{\mathbf{e}}}_\theta = -\dot{\theta} \hat{\mathbf{e}}_r$$

$$\vec{\mathbf{v}} = \dot{r} \hat{\mathbf{e}}_r + r \dot{\theta} \hat{\mathbf{e}}_\theta$$

$$\vec{\mathbf{a}} = \left(\ddot{r} - r \dot{\theta}^2 \right) \hat{\mathbf{e}}_r + \left(r \ddot{\theta} + 2 \dot{r} \dot{\theta} \right) \hat{\mathbf{e}}_\theta$$

Differentiating Vectors Expressed in Rotating Frame

If the position vector $\vec{\mathbf{r}}$ is expressed in a reference frame that has angular velocity $\vec{\omega}$ with respect to an inertial reference frame, then the components of the velocity vector $\dot{\vec{\mathbf{r}}}$ expressed in the rotating frame are found using

$$\dot{\vec{\mathbf{r}}} + \omega^\times \mathbf{r}$$

where ω^\times denotes the 3×3 skew-symmetric matrix

$$\omega^\times = \begin{bmatrix} 0 & -\omega_3 & \omega_2 \\ \omega_3 & 0 & -\omega_1 \\ -\omega_2 & \omega_1 & 0 \end{bmatrix}$$

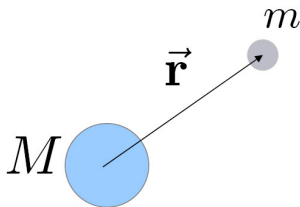
Differentiating again gives the components of the acceleration vector, $\ddot{\vec{\mathbf{r}}}$, expressed in the rotating reference frame:

$$\ddot{\vec{\mathbf{r}}} + 2\omega^\times \dot{\vec{\mathbf{r}}} + \dot{\omega}^\times \mathbf{r} + \omega^\times (\omega^\times \mathbf{r})$$

The first two equations on this slide will be used throughout this course in using vectors expressed in rotating reference frames.

Orbital Mechanics Application

- When the net force is easily decomposed into radial and transverse components, the preceding system is quite useful
- The most common example is the simple two-body problem, where the force is the inverse-square universal gravitational law



$$\vec{\mathbf{f}} = -\frac{GMm}{r^2}\hat{\mathbf{e}}_r$$

where G is the universal gravitational constant

Suppose we select units such that $GM=1$; then the force is

$$\vec{\mathbf{f}}_r = -\frac{m}{r^2}\hat{\mathbf{e}}_r$$

Application of Newton's 2nd law leads to the two scalar nonlinear ordinary differential equations:

$$\ddot{r} - r\dot{\theta}^2 = -\frac{1}{r^2}$$

$$r\ddot{\theta} + 2\dot{r}\dot{\theta} = 0$$

Why are these equations *nonlinear*?

Simple Orbital Example (continued)

Rewrite the nonlinear differential equations as $\dot{\mathbf{x}} = \mathbf{f}(\mathbf{x})$, where $\mathbf{x} \in \mathbf{R}^4$, the set of 4×1 real matrices.

Define new state variables:

$$\begin{array}{lll} x_1 & = & r \\ x_2 & = & \theta \\ x_3 & = & \dot{r} \\ x_4 & = & \dot{\theta} \\ \downarrow & & \\ \mathbf{x} & \in & \mathbf{R}^4 \end{array} \quad \Rightarrow \quad \begin{array}{ll} \dot{x}_1 & = x_3 \\ \dot{x}_2 & = x_4 \\ \dot{x}_3 & = x_1 x_4^2 - \frac{1}{x_1^2} \\ \dot{x}_4 & = -\frac{2x_3 x_4}{x_1} \\ \downarrow & \\ \dot{\mathbf{x}} & = \mathbf{f}(\mathbf{x}) \end{array}$$

Note that the state differential equation is a 4×1 system of coupled, nonlinear, ordinary differential equations.

Simple Orbital Example (continued)

- ▶ We can identify a special solution to the equations of motion for circular orbits
- ▶ Suppose $x_1 = \text{constant}$ (circular orbit), then $x_3 = 0$, and $x_4 = \text{constant}$ (constant angular rate, or mean motion)
- ▶ From the differential equation for x_3 , which equals zero in this case, we can deduce that

$$x_1^3 x_4^2 = 1$$

- ▶ This result is a form of Kepler's Third Law, which relates the orbital period to the orbital radius
- ▶ The "1" is a result of the choice of canonical units with gravitational parameter $\mu = 1$

Orbital Mechanics Integrator

```
%(name file TwoBodyExampleDriver.m)
% Two-Body Example Driver

x1=1; x2=0; x3=0; x4=1;           % initial conditions
x0 = [x1; x2; x3; x4];
tspan= linspace(0,2*pi,100);      % circular orbit period is 2pi

% setting up to use ode45
options = odeset('abstol',1e-8, 'reltol',1e-8);
rhsfn = 'TwoBodyExa';

% calling ode45 to do integration
[t,x] = ode45(rhsfn,tspan,x0,options);

% graph results
figure; hold on
hg=plot(x(:,1).*cos(x(:,2)), x(:,1).*sin(x(:,2)));
set(hg,'linestyle','-','linewidth',2);
```

Orbital Mechanics Integrator (2)

```
% changing dtheta/dt initial condition, period not 2pi
x0(4)= 1.1;
E=(x0(1)*x0(4))^2/2 - 1/x0(1);
a=-1/(2*E);
TP=2*pi*sqrt(a^3/1);
tspan=linspace(0,TP,100);
[t,x] = ode45(rhsfn,tspan,x0,options);
hg=plot(x(:,1).*cos(x(:,2)), x(:,1).*sin(x(:,2)));
set(hg,'linestyle',':', 'linewidth',2);
x0(4)= 0.8;
E=(x0(1)*x0(4))^2/2 - 1/x0(1); a=-1/(2*E);
TP=2*pi*sqrt(a^3/1); tspan=linspace(0,TP,100);
[t,x] = ode45(rhsfn,tspan,x0,options);
hg=plot(x(:,1).*cos(x(:,2)), x(:,1).*sin(x(:,2)));
set(hg,'linestyle','--', 'linewidth',2);
hg=xlabel('x = r cos \theta'); set(hg,'fontsize',18);
hg=ylabel('y = r sin \theta'); set(hg,'fontsize',18);
legend('Circular','Large Ellipse','Small Ellipse');
set(gca,'fontsize',18);

% END OF TwoBodyExampleDriver.m
```

Orbital Mechanics Integrator (3)

The Matlab script `TwoBodyExampleDriver` calls the built-in function `ode45` and the following “right-hand side” function `TwoBodyExa`.

The “right-hand side” function simply returns the 4×1 matrix of the values of $f(\mathbf{x})$ calculated with the current value of \mathbf{x} .

```
% (name file TwoBodyExa.m)
function xdot = TwoBodyExa(t,x,options)
xdot=[x(3);
x(4);
x(1)*x(4)^2-1/x(1)^2;
-2*x(3)*x(4)/x(1)];
```

Using `TwoBodyExampleDriver` and `TwoBodyExa`, we can integrate the equations of motion and generate graphs of the orbits.

This Matlab file structure can be used to set up numerical integration for other systems of differential equations.

Orbital Mechanics Integrator (4)

