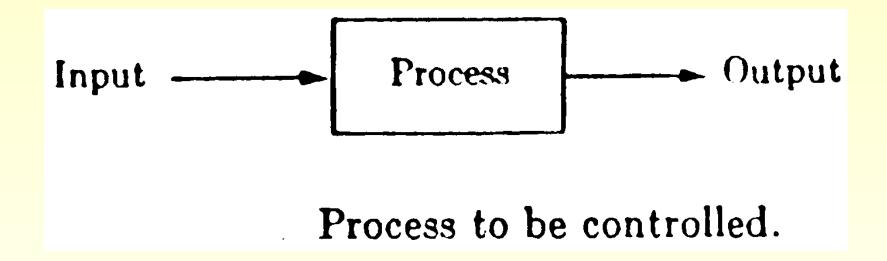
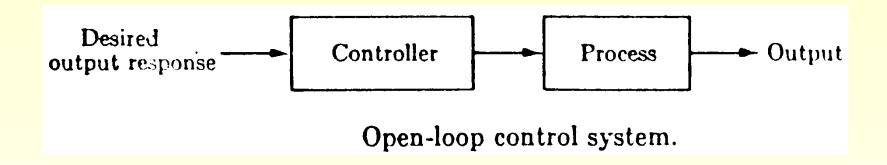
Closed loop systems

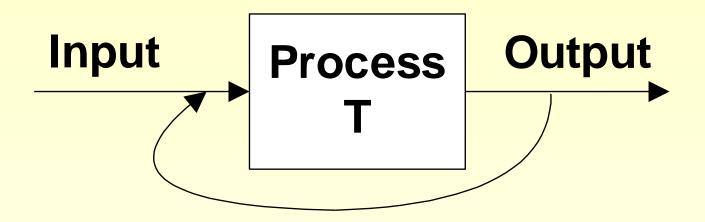
Process



Open loop control system



Closed system



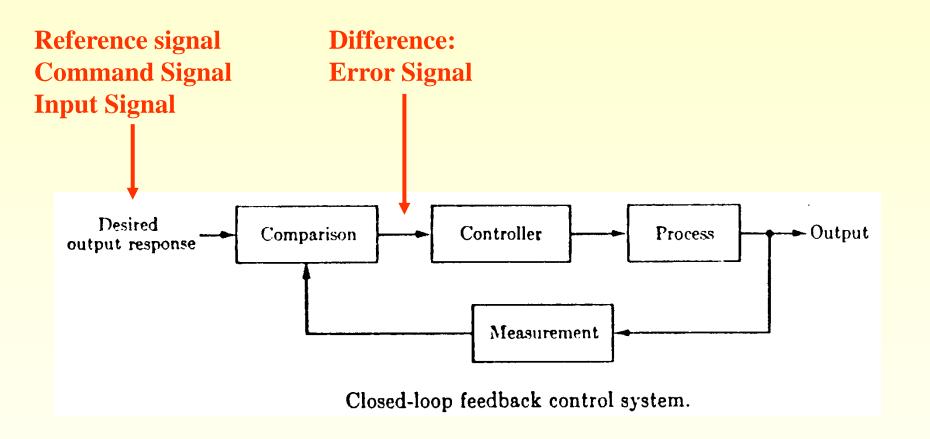
Negative feedback: stable levels

E.g., thermoregulation

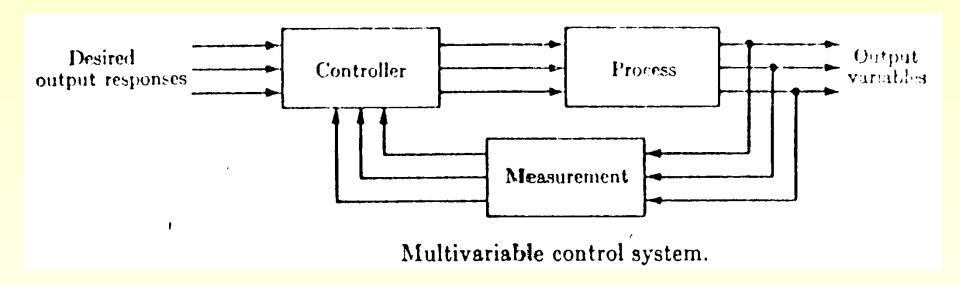
Positive feedback: on-off

E.g. action potential firing, cell division, cell death, blood coagulation, micturition

Closed loop feedback system

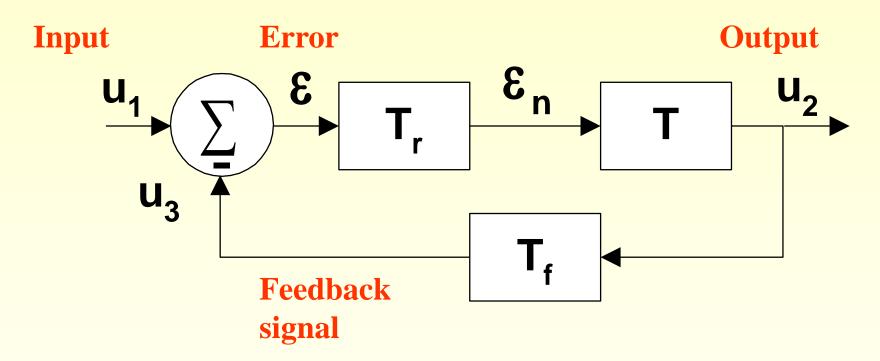


Closed loop feedback system with multiple inputs and outputs



Representation of a control system

Comparator



Types of Regulators

- > Continuous
- Proportional (P) $\varepsilon_n = k\varepsilon$
- Differentiating (D) $\varepsilon_n = k \frac{d\varepsilon}{dt}$
- Integrating (I) $\varepsilon_n = k \int \varepsilon dt$
- Combinations (PD, PI, PID)

Types of Regulators

- > Discontinuous
- Two or more states: Can have only a finite number of different values dependent on the value of the error. E.g. thermostat.
- Constant velocity: $d\epsilon_n/dt$ can only have two different values: 0 or 1

Properties of control systems

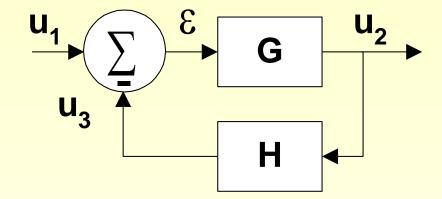
$$u_2 = G\varepsilon$$

$$\varepsilon = u_1 - u_3$$

$$u_3 = H u_2$$

$$\varepsilon = \frac{1}{1 + GH} u_1$$

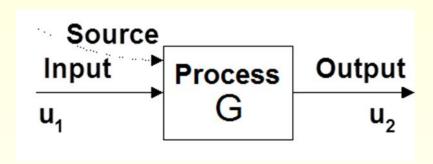
$$u_2 = \frac{G}{1 + GH} u_1$$



The gain of a closed system is smaller than that of the open system

Sensitivity analysis Definition of sensitivity

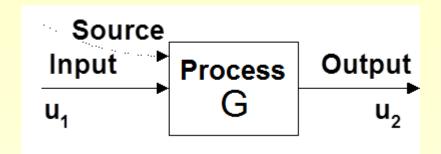
$$S_x = \frac{\frac{\delta u_2}{u_2}}{\frac{\delta x}{x}}$$



Sensitivity of open system to G

$$u_2 = Gu_1 \rightarrow \delta u_2 = \delta Gu_1$$

$$\frac{\delta u_2}{u_2} = \frac{\delta G u_1}{u_2} = \frac{\delta G u_1}{G u_1} = \frac{\delta G}{G}$$



$$S_G = \frac{\frac{\delta u_2}{u_2}}{\frac{\delta G}{G}} = 1$$

High sensitivity to variations in the process G

Sensitivity of closed system to G

$$u_2 = \frac{G}{1 + GH}u_1$$

$$\delta u_2 = \frac{\delta G(1+GH) - GH\delta G}{(1+GH)^2} u_1$$

$$\begin{array}{c} U_1 \\ \downarrow \\ U_3 \end{array} \qquad \begin{array}{c} E \\ \downarrow \\ H \end{array} \qquad \begin{array}{c} U_2 \\ \downarrow \\ \end{array}$$

$$\delta u_2 = \frac{\delta G}{(1 + GH)^2} u_1$$

$$\frac{\delta u_2}{u_2} = \frac{1}{1 + GH} \frac{\delta G}{G}$$

$$S_G = \frac{1}{1 + GH}$$

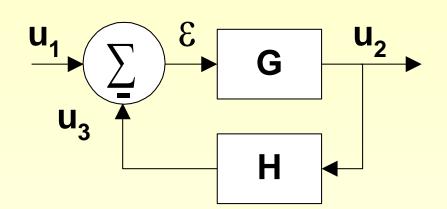
for GH >> 1
$$\Rightarrow$$
 $S_G \cong 1/GH$

Low sensitivity to variations in the process G

Sensitivity of closed system to H

$$u_2 = \frac{G}{1 + GH} u_1$$

$$\delta u_2 = \frac{-G^2 \delta H}{(1 + GH)^2} u_1$$



$$\frac{\delta u_2}{u_2} = \frac{-G}{1 + GH} \delta H = \frac{-GH}{1 + GH} \frac{\delta H}{H}$$

$$S_H = \frac{-GH}{1 + GH}$$

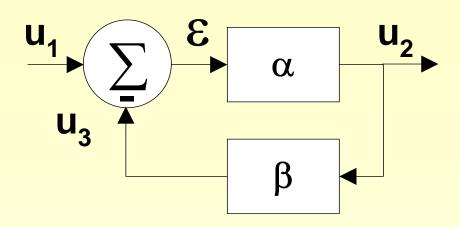
for GH >> 1
$$\Rightarrow$$
 $S_H \cong -1$

High sensitivity to variations in H, but this process is typically a transmission line and can be made simple to reduce errors

Proportional feedback. Zero order process

$$u_2 = \frac{\alpha}{1 + \alpha\beta} u_1$$

$$S_{\alpha} = \frac{1}{1 + \alpha \beta}$$



For $\beta = 1$: Follower system.

$$u_2 = u_1/(1 + 1/\alpha)$$

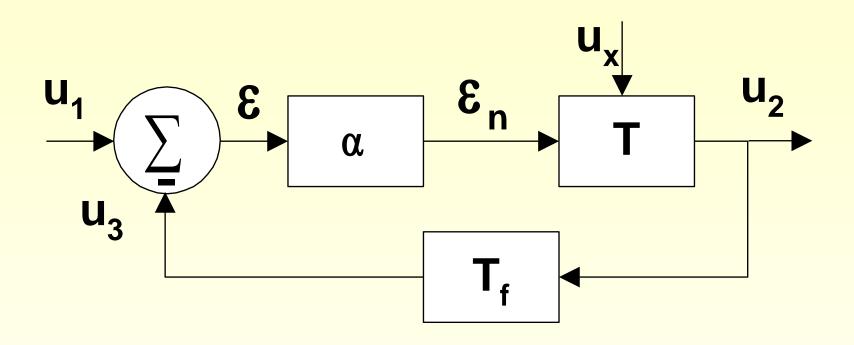
$$\alpha \to \infty \implies u_2 \to u_1$$

Steady state error:

$$\Delta = (u_2 - u_1)/u_1 = -1/(1 + \alpha)$$
Approximation:

$$\Delta \cong -1/\alpha$$
 for $\alpha >> 1$

First order process (with disturbance)



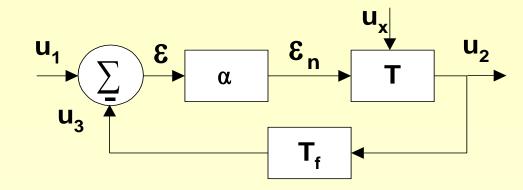
First order process (with disturbance)

Open system (1)

$$\tau \frac{du_2}{dt} + u_2 = u_1 + u_x$$

Open system with amplification (2)

$$\tau \frac{du_2}{dt} + u_2 = \alpha u_1 + u_x$$



Closed system (3)

$$\tau \frac{du_2}{dt} + u_2 = \varepsilon_n + u_x$$
$$\varepsilon_n = \alpha \varepsilon \qquad \varepsilon = u_1 - u_3$$

$$\frac{\tau}{1+\alpha}\frac{du_2}{dt} + u_2 = \frac{\alpha}{1+\alpha}u_1 + \frac{1}{1+\alpha}u_x$$

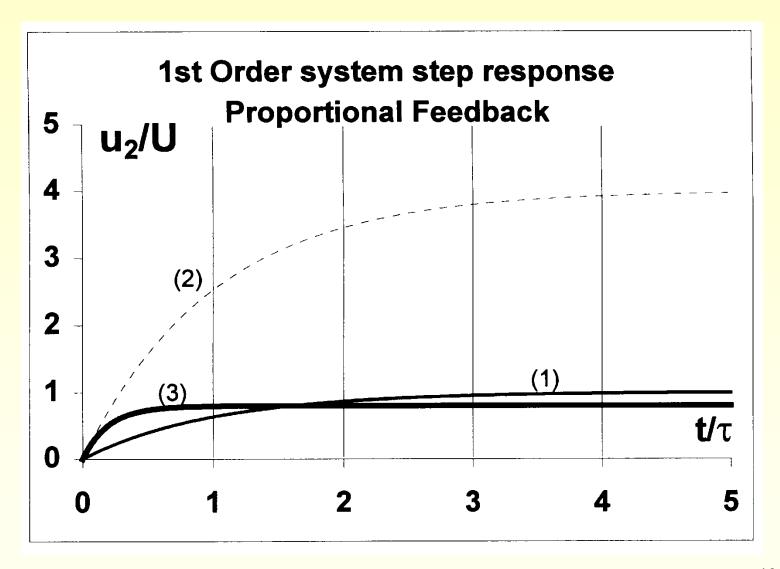
First order process (with disturbance)

$$\tau \frac{du_2}{dt} + u_2 = u_1 + u_x \qquad \frac{\tau}{1 + \alpha} \frac{du_2}{dt} + u_2 = \frac{\alpha}{1 + \alpha} u_1 + \frac{1}{1 + \alpha} u_x$$

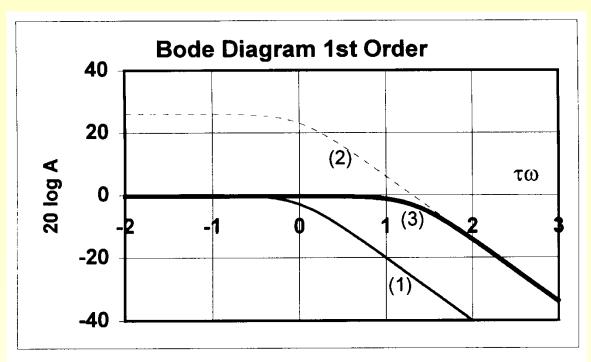
$$\tau \frac{du_2}{dt} + u_2 = \alpha u_1 + u_x$$

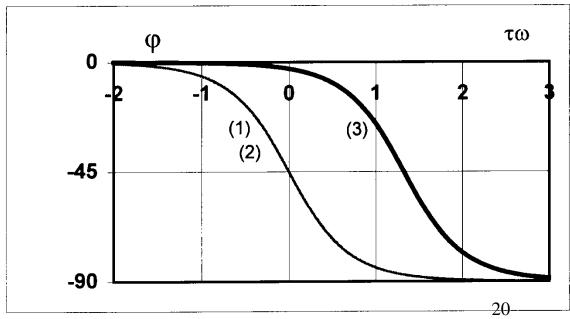
	System 1	System 2	System 3
Time constant τ'	τ	τ	$\tau/(1+\alpha)$
Gain α'	1	α	$\alpha/(1+\alpha)$
Gain of disturbance α' _x	1	1	1/(1+α)

First order process

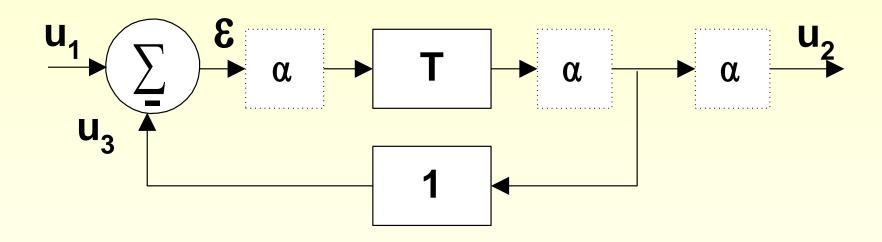


Bode diagram first order system





Extra amplification between comparator and regulator to compensate for the reduced gain



Second order process

Open system:

$$\frac{1}{(\omega_n)^2} \frac{d^2 u_2}{dt^2} + \frac{2\xi}{\omega_n} \frac{du_2}{dt} + u_2 = u_1$$

Closed system (with extra gain):

$$\frac{1}{(\omega_n)^2} \frac{d^2 u_2}{dt^2} + \frac{2\xi}{\omega_n} \frac{du_2}{dt} + u_2 = \alpha \varepsilon$$

$\varepsilon = u_1 - u_2$ (we assume unity gain feedback):

$$\frac{1}{(1+\alpha)(\omega_n)^2} \frac{d^2 u_2}{dt^2} + \frac{2\xi}{(1+\alpha)\omega_n} \frac{du_2}{dt} + u_2 = \frac{\alpha}{1+\alpha} u_1$$

2nd order process with unit feedback Properties

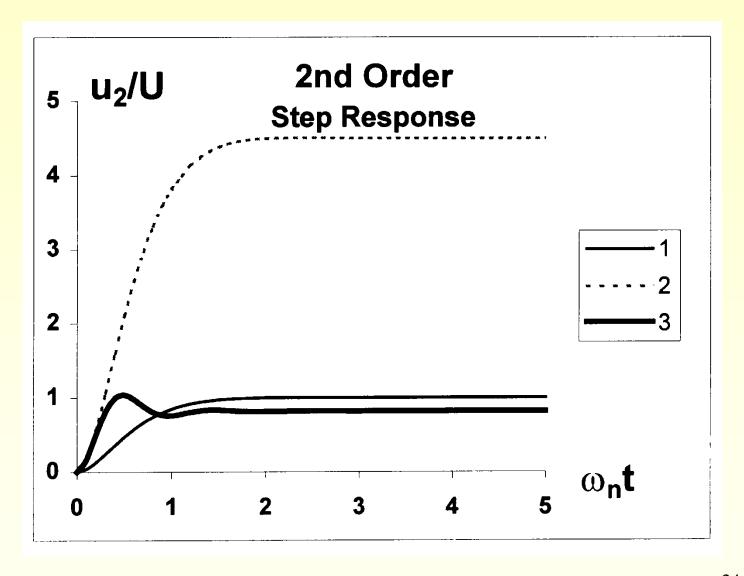
$$\omega'_{n} = \omega_{n} \cdot \sqrt{1+\alpha}$$

$$\xi' = \xi/\sqrt{1+\alpha}$$

$$\alpha' = \alpha/(1+\alpha)$$

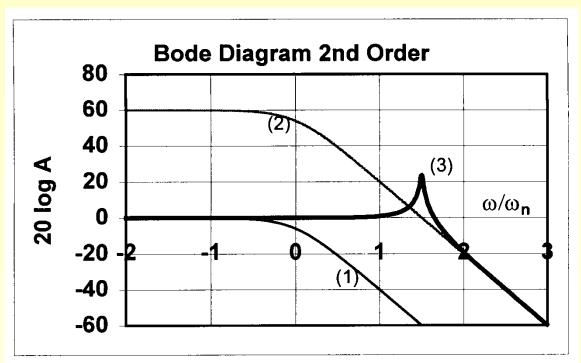
When α is high the damping factor decreases and there is risk of undamped oscillations

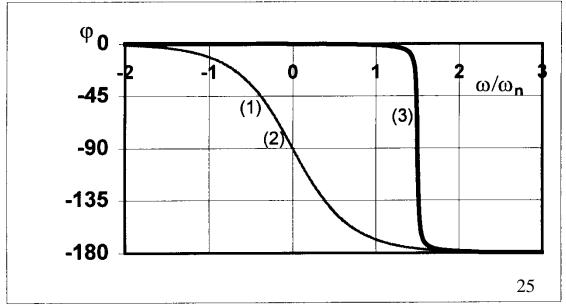
2nd order feedback process Step response



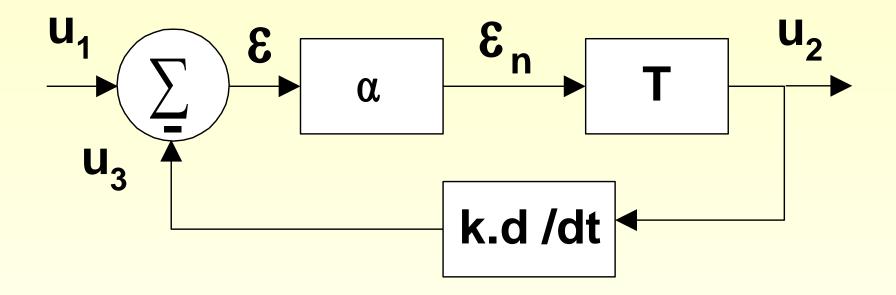
1 Open

2nd order feedback process. Bode diagram

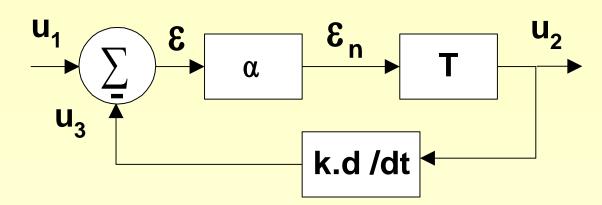




Differentiating feedback



Differentiating feedback: Parameters



$$\frac{1}{(\omega_n)^2} \frac{d^2 u_2}{dt^2} + \frac{2\xi}{\omega_n} \frac{du_2}{dt} + u_2 = \alpha \varepsilon$$

$$\varepsilon = u_1 - u_3 \qquad u_3 = k \frac{du_2}{dt}$$

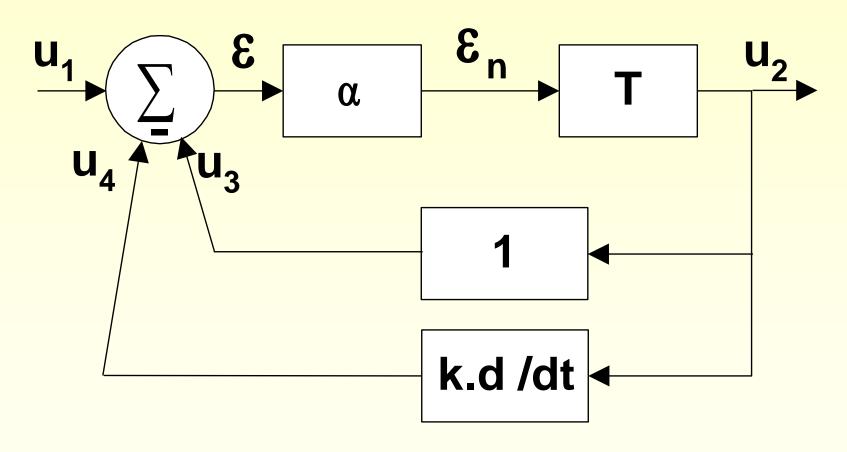
$$\frac{1}{(\omega_n)^2} \frac{d^2 u_2}{dt^2} + \frac{2(\xi + \omega_n \alpha k/2)}{\omega_n} \cdot \frac{du_2}{dt} + u_2 = \alpha u_1$$

$$\omega'_n = \omega_n$$

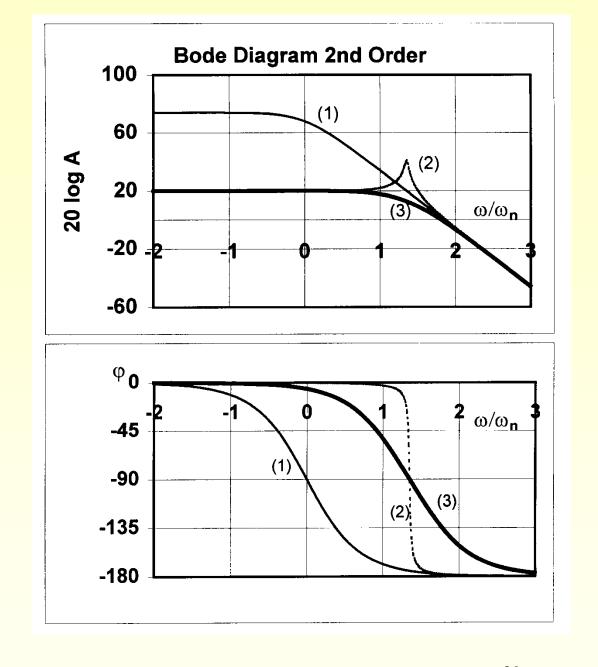
$$\xi' = \xi + \omega_n \alpha k/2$$

$$\alpha' = \alpha$$

Proportional feedback with differentiating correction



Proportional feedback with differentiating correction
Bode diagram



1 Open 2 Proportional feedback 3 Proport. feedback with diff. corr.

Integrating feedback

$$\frac{1}{(\omega_n)^2} \frac{d^2 u_2}{dt^2} + \frac{2\xi}{\omega_n} \frac{du_2}{dt} + u_2 = \alpha \varepsilon$$

$$\varepsilon = u_1 - u_3$$
 $u_3 = k \int u_2 dt$

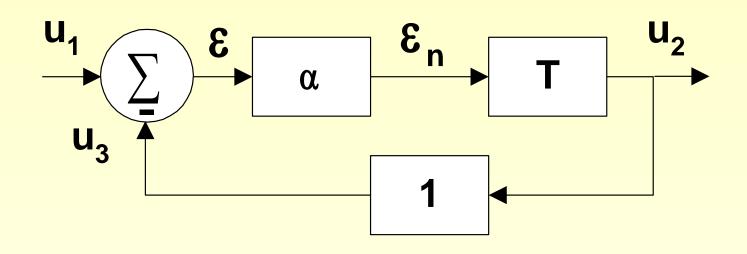
$$\frac{1}{(\omega_n)^2} \frac{d^2 u_2}{dt^2} + \frac{2\xi}{\omega_n} \frac{du_2}{dt} + u_2 + \alpha k \int u_2 dt = \alpha u_1$$

Differentiation of both sides of the equation:

$$\frac{1}{(\omega_n)^2} \frac{d^3 u_2}{dt^3} + \frac{2\xi}{\omega_n} \frac{d^2 u_2}{dt^2} + \frac{du_2}{dt} + \alpha k u_2 = \alpha \frac{du_1}{dt}$$

Stability of feedback systems

Stability of feedback systems



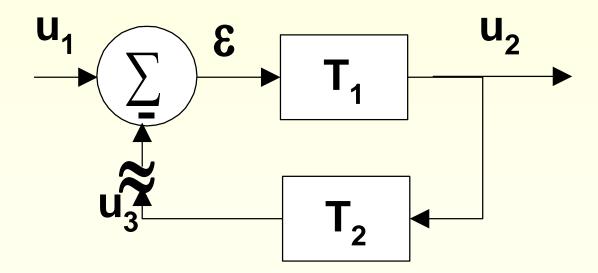
With T a system of order > 2

There is a frequency ω_k for which $\phi_{u_2}(\omega_k) = -180^\circ$ $u_2 = A_2 \cdot \sin(\omega_k t - 180^\circ) = -A_2 \cdot \sin(\omega_k t)$ $u_3 = u_2$ $\epsilon = u_1 - u_3$ $\epsilon = A_1 \cdot \sin(\omega_k t) - (-A_2 \cdot \sin(\omega_k t))$ $\epsilon = (A_1 + A_2) \cdot \sin(\omega_k t) > A_1 \cdot \sin(\omega_k t)$

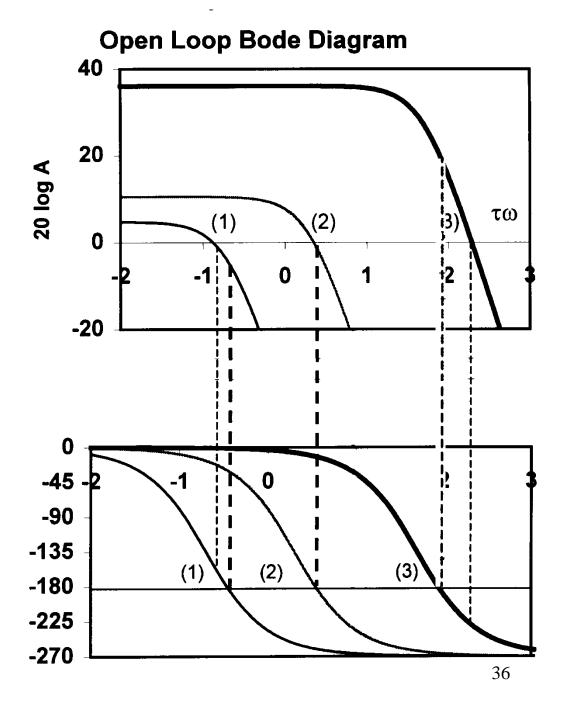
Nyquist criterion

A system with negative feedback is unstable if the open loop gain is larger than or equal to one at the frequency at which the open loop phase shift is $= 180^{\circ}$

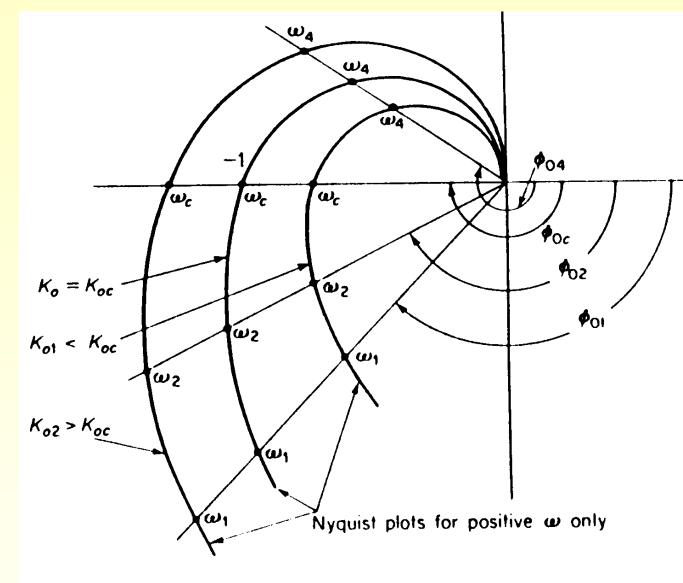
Open loop transfer function: T₁T₂



Nyquist criterion in Bode diagram

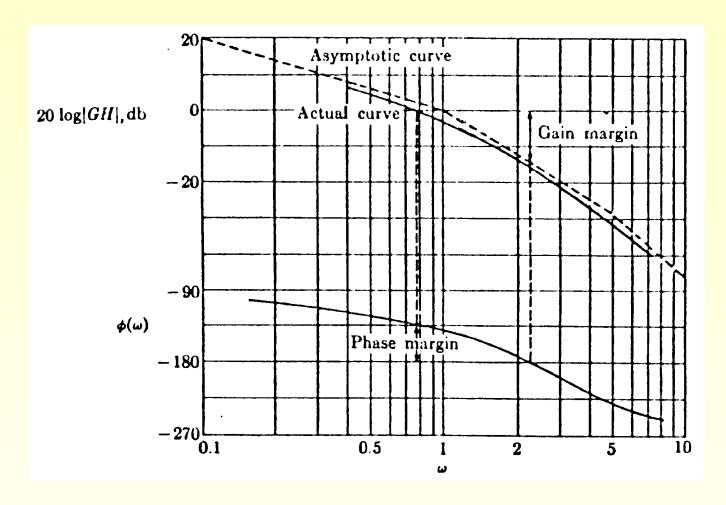


Nyquist criterion in polar plot



Effect of K_o on Nyquist plots of the form

Gain and phase margin



Gain and phase shifts that can be introduced without the system becoming unstable