

Input Response

Control Theory, Lecture 6

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STEADY-STATE GAIN: ODE

Given an ODE with a constant input $c = \text{const}$:

$$a_n y^{(n)} + \dots + a_1 \dot{y} + a_0 y = b_m u^{(m)} + \dots + b_1 \dot{u} + b_0 u \quad (1)$$

$$u(t) = c \quad (2)$$

This is equivalent to:

$$a_n y^{(n)} + \dots + a_1 \dot{y} + a_0 y = b_0 c \quad (3)$$

A steady-state solution $y_{ss} = \text{const}$:

$$a_0 y_{ss} = b_0 c \quad (4)$$

$$y_{ss} = \frac{b_0}{a_0} c \quad (5)$$

The quantity $K = \frac{b_0}{a_0}$ is the steady-state gain of the system.

STEADY-STATE GAIN: TRANSFER FUNCTION

Assume the system \mathcal{G} represented as a transfer function:

$$Y(s) = \frac{b_m s^m + \dots + b_0}{a_n s^n + \dots + a_0} U(s) \quad (6)$$

Then, as any element multiplied by the differential operator s with power higher than 0 is a derivative of u or y and both are 0 at the steady-state solution, the steady-state gain can be found by setting those to zero:

$$K = \frac{b_0}{a_0} \quad (7)$$

STEADY-STATE GAIN: STATE SPACE, 1

Given an LTI with a constant input $\mathbf{u}_{ss} = \text{const}$:

$$\begin{cases} \dot{\mathbf{x}} = \mathbf{A}\mathbf{x} + \mathbf{B}\mathbf{u} \\ \mathbf{y} = \mathbf{C}\mathbf{x} \end{cases} \quad (8)$$

A steady-state solution $\mathbf{x}_{ss} = \text{const}$:

$$\begin{cases} 0 = \mathbf{A}\mathbf{x}_{ss} + \mathbf{B}\mathbf{u}_{ss} \\ \mathbf{y}_{ss} = \mathbf{C}\mathbf{x}_{ss} \end{cases} \quad (9)$$

$$\mathbf{y}_{ss} = -\mathbf{C}\mathbf{A}^{-1}\mathbf{B}\mathbf{u}_{ss} \quad (10)$$

The quantity $K = -\mathbf{C}\mathbf{A}^{-1}\mathbf{B}$ is the steady-state gain of the system.

STEADY-STATE GAIN: STATE SPACE, 2

For the following LTI:

$$\begin{cases} \begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \\ \dot{x}_3 \end{bmatrix} = \begin{bmatrix} 0 & 1 & 0 \\ 0 & 0 & 1 \\ -a_0 & -a_1 & -a_2 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix} + \begin{bmatrix} 0 \\ 0 \\ b_0 \end{bmatrix} \mathbf{u} \\ \mathbf{y} = \begin{bmatrix} 1 & 0 & 0 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix} \end{cases} \quad (11)$$

Then we get:

$$\mathbf{y}_{ss} = - \begin{bmatrix} 1 & 0 & 0 \end{bmatrix} \begin{bmatrix} -a_1/a_0 & -a_2/a_0 & -1/a_0 \\ 1 & 0 & 0 \\ 0 & 1 & 0 \end{bmatrix} \begin{bmatrix} 0 \\ 0 \\ b_0 \end{bmatrix} \mathbf{u}_{ss} \quad (12)$$

The quantity $K = -\mathbf{CA}^{-1}\mathbf{B} = \frac{b_0}{a_0}$ is the steady-state gain of the system.

FREQUENCY RESPONSE, 1

Consider LTI with input $\mathbf{u} = \alpha \cos(\omega t) + \beta \sin(\omega t)$:

$$\begin{cases} \dot{\mathbf{x}} = \mathbf{A}\mathbf{x} + \mathbf{B}\mathbf{u} \\ \mathbf{y} = \mathbf{C}\mathbf{x} \end{cases} \quad (13)$$

with steady-state solution:

$$x_i = g_i \cos(\omega t) + h_i \sin(\omega t) \quad (14)$$

$$\dot{x}_i = \omega h_i \cos(\omega t) - \omega g_i \sin(\omega t) \quad (15)$$

$$y_i = q \cos(\omega t) + r \sin(\omega t) \quad (16)$$

In the vector form:

$$\begin{bmatrix} x_1 \\ \vdots \\ x_n \end{bmatrix} = \begin{bmatrix} g_1 \\ \vdots \\ g_n \end{bmatrix} \cos(\omega t) + \begin{bmatrix} h_1 \\ \vdots \\ h_n \end{bmatrix} \sin(\omega t) \quad (17)$$

$$\mathbf{x} = \mathbf{g} \cos(\omega t) + \mathbf{h} \sin(\omega t) \quad (18)$$

and: $\dot{\mathbf{x}} = \omega \mathbf{h} \cos(\omega t) - \omega \mathbf{g} \sin(\omega t)$

Thus:

$$\mathbf{u} = \alpha \cos(\omega t) + \beta \sin(\omega t) \quad (19)$$

$$\mathbf{x} = \mathbf{g} \cos(\omega t) + \mathbf{h} \sin(\omega t) \quad (20)$$

$$\dot{\mathbf{x}} = \omega \mathbf{h} \cos(\omega t) - \omega \mathbf{g} \sin(\omega t) \quad (21)$$

With that we can re-write $\dot{\mathbf{x}} = \mathbf{A}\mathbf{x} + \mathbf{B}\mathbf{u}$ as:

$$\begin{aligned} \omega \mathbf{h} \cos(\omega t) - \omega \mathbf{g} \sin(\omega t) &= \\ &= \mathbf{A}\mathbf{g} \cos(\omega t) + \mathbf{A}\mathbf{h} \sin(\omega t) + \mathbf{B}\alpha \cos(\omega t) + \mathbf{B}\beta \sin(\omega t) \end{aligned}$$

Grouping terms in front of cosines we get:

$$\omega \mathbf{h} = \mathbf{A}\mathbf{g} + \mathbf{B}\alpha \quad (22)$$

And grouping terms in front of sines :

$$-\omega \mathbf{g} = \mathbf{A}\mathbf{h} + \mathbf{B}\beta \quad (23)$$

We study the equation $\mathbf{y} = \mathbf{C}\mathbf{x}$ in the same way:

$$q \cos(\omega t) + r \sin(\omega t) = \mathbf{C}\mathbf{g} \cos(\omega t) + \mathbf{C}\mathbf{h} \sin(\omega t) \quad (24)$$

Grouping terms in front of cosines we get:

$$q = \mathbf{C}\mathbf{g} \quad (25)$$

And grouping terms in front of sines :

$$r = \mathbf{C}\mathbf{h} \quad (26)$$

We have the following equations:

$$\omega \mathbf{h} = \mathbf{A} \mathbf{g} + \mathbf{B} \alpha \quad (27)$$

$$-\omega \mathbf{g} = \mathbf{A} \mathbf{h} + \mathbf{B} \beta \quad (28)$$

$$q = \mathbf{C} \mathbf{g} \quad (29)$$

$$r = \mathbf{C} \mathbf{h} \quad (30)$$

This can be re-written in a matrix form:

$$\begin{cases} \begin{bmatrix} -\mathbf{A} & \omega \mathbf{I} \\ -\omega \mathbf{I} & -\mathbf{A} \end{bmatrix} \begin{bmatrix} \mathbf{g} \\ \mathbf{h} \end{bmatrix} = \begin{bmatrix} \mathbf{B} & 0 \\ 0 & \mathbf{B} \end{bmatrix} \begin{bmatrix} \alpha \\ \beta \end{bmatrix} \\ \begin{bmatrix} q \\ r \end{bmatrix} = \begin{bmatrix} \mathbf{C} & 0 \\ 0 & \mathbf{C} \end{bmatrix} \begin{bmatrix} \mathbf{g} \\ \mathbf{h} \end{bmatrix} \end{cases} \quad (31)$$

This system can be written as:

$$\begin{cases} \begin{bmatrix} -\mathbf{A} & \omega\mathbf{I} \\ -\omega\mathbf{I} & -\mathbf{A} \end{bmatrix} \begin{bmatrix} \mathbf{g} \\ \mathbf{h} \end{bmatrix} = \begin{bmatrix} \mathbf{B} & 0 \\ 0 & \mathbf{B} \end{bmatrix} \begin{bmatrix} \alpha \\ \beta \end{bmatrix} \\ \begin{bmatrix} q \\ r \end{bmatrix} = \begin{bmatrix} \mathbf{C} & 0 \\ 0 & \mathbf{C} \end{bmatrix} \begin{bmatrix} \mathbf{g} \\ \mathbf{h} \end{bmatrix} \end{cases} \quad (32)$$

Note that we can compute the steady-state solution for all states of this system:

$$\begin{bmatrix} \mathbf{g} \\ \mathbf{h} \end{bmatrix} = \begin{bmatrix} -\mathbf{A} & \omega\mathbf{I} \\ -\omega\mathbf{I} & -\mathbf{A} \end{bmatrix}^{-1} \begin{bmatrix} \mathbf{B} & 0 \\ 0 & \mathbf{B} \end{bmatrix} \begin{bmatrix} \alpha \\ \beta \end{bmatrix} \quad (33)$$

We can also solve for the steady-state output:

$$\begin{bmatrix} q \\ r \end{bmatrix} = \begin{bmatrix} \mathbf{C} & 0 \\ 0 & \mathbf{C} \end{bmatrix} \begin{bmatrix} -\mathbf{A} & \omega\mathbf{I} \\ -\omega\mathbf{I} & -\mathbf{A} \end{bmatrix}^{-1} \begin{bmatrix} \mathbf{B} & 0 \\ 0 & \mathbf{B} \end{bmatrix} \begin{bmatrix} \alpha \\ \beta \end{bmatrix} \quad (34)$$

The map between the input and the output as:

$$\mathbf{M}(\omega) = \begin{bmatrix} \mathbf{C} & 0 \\ 0 & \mathbf{C} \end{bmatrix} \begin{bmatrix} -\mathbf{A} & \omega \mathbf{I} \\ -\omega \mathbf{I} & -\mathbf{A} \end{bmatrix}^{-1} \begin{bmatrix} \mathbf{B} & 0 \\ 0 & \mathbf{B} \end{bmatrix} \quad (35)$$

We can define input coordinates $\zeta = \begin{bmatrix} \alpha \\ \beta \end{bmatrix}$ and output

coordinates $\mathbf{p} = \begin{bmatrix} q \\ r \end{bmatrix} = \mathbf{M}\zeta$. The amplitude amplification can be defined as the ratio:

$$\text{amp}(\omega) = \frac{\sqrt{q^2 + r^2}}{\sqrt{\alpha^2 + \beta^2}} = \frac{\|\mathbf{p}\|}{\|\zeta\|} = \frac{\|\mathbf{M}\zeta\|}{\|\zeta\|} \quad (36)$$

Notice that by definition $\max_{\zeta} \frac{\|\mathbf{M}\zeta\|_2}{\|\zeta\|_2} = \|\mathbf{M}\|_2 = \sigma_{\max}(\mathbf{M})$, where $\sigma_{\max}(\mathbf{M})$ is the largest singular value of the matrix.

The phase shift can be defined as:

$$\text{phase}(\omega) \sim \angle(\mathbf{p}) - \angle(\zeta) = \angle(\mathbf{M}\zeta) - \angle(\zeta) \quad (37)$$

But matrix \mathbf{M} is a scaled orthonormal matrix:

$$\mathbf{M} = \text{amp}(\omega) \begin{bmatrix} \cos(\varphi) & -\sin(\varphi) \\ \sin(\varphi) & \cos(\varphi) \end{bmatrix} \quad (38)$$

Thus:

$$\text{phase}(\omega) \sim \varphi = \text{atan2}(M_{21}, M_{11}) \quad (39)$$

MIMO FREQUENCY RESPONSE, 1

Transfer function-based Bode plot relies on a SISO representation of a system. However, choosing input and output one can use it for a MIMO system as well.

State-space representation naturally points out the connection between inputs and outputs and the resulting response. Consider a fixed input matrix $\mathbf{B} \in \mathbb{R}^{n \times 1}$; we can ask a question, what choice of the output $\mathbf{C} \in \mathbb{R}^{1 \times n}$ (assuming $\|\mathbf{C}\| = 1$) produces the largest amplitude of the output signal.

MIMO FREQUENCY RESPONSE, 2

From the previous slides we saw that:

$$\text{amp}(\omega) \begin{bmatrix} \cos(\varphi) & -\sin(\varphi) \\ \sin(\varphi) & \cos(\varphi) \end{bmatrix} = \begin{bmatrix} \mathbf{C} & 0 \\ 0 & \mathbf{C} \end{bmatrix} \begin{bmatrix} -\mathbf{A} & \omega \mathbf{I} \\ -\omega \mathbf{I} & -\mathbf{A} \end{bmatrix}^{-1} \begin{bmatrix} \mathbf{B} & 0 \\ 0 & \mathbf{B} \end{bmatrix}$$

Defining:

$$\begin{bmatrix} \mathbf{P}_{11} & \mathbf{P}_{12} \\ \mathbf{P}_{21} & \mathbf{P}_{22} \end{bmatrix} = \begin{bmatrix} -\mathbf{A} & \omega \mathbf{I} \\ -\omega \mathbf{I} & -\mathbf{A} \end{bmatrix}^{-1} \begin{bmatrix} \mathbf{B} & 0 \\ 0 & \mathbf{B} \end{bmatrix}, \quad \text{amp}(\omega) = m \quad (40)$$

We find:

$$\begin{bmatrix} m \cos(\varphi) & -m \sin(\varphi) \\ m \sin(\varphi) & m \cos(\varphi) \end{bmatrix} = \begin{bmatrix} \mathbf{C}\mathbf{P}_{11} & \mathbf{C}\mathbf{P}_{12} \\ \mathbf{C}\mathbf{P}_{21} & \mathbf{C}\mathbf{P}_{22} \end{bmatrix} \quad (41)$$

$$m^2 \cos^2(\varphi) + m^2 \sin^2(\varphi) = \mathbf{C}\mathbf{P}_{11}\mathbf{P}_{11}^\top \mathbf{C}^\top + \mathbf{C}\mathbf{P}_{21}\mathbf{P}_{21}^\top \mathbf{C}^\top \quad (42)$$

$$m^2 = \mathbf{C}(\mathbf{P}_{11}\mathbf{P}_{11}^\top + \mathbf{P}_{21}\mathbf{P}_{21}^\top) \mathbf{C}^\top \quad (43)$$

MIMO FREQUENCY RESPONSE, 3

Defining $\mathbf{N} = \mathbf{P}_{11}\mathbf{P}_{11}^\top + \mathbf{P}_{21}\mathbf{P}_{21}^\top$ with decomposition $\mathbf{N} = \mathbf{D}\mathbf{D}^\top$ we can find the maximum value of m by maximizing:

$$m = \max_{\mathbf{C}} \frac{\sqrt{\mathbf{C}\mathbf{D}\mathbf{D}^\top\mathbf{C}^\top}}{\|\mathbf{C}\|} = \max_{\mathbf{C}} \frac{\|\mathbf{D}^\top\mathbf{C}^\top\|}{\|\mathbf{C}\|} = \sigma_{\max}(\mathbf{D}) = \sqrt{\sigma_{\max}(\mathbf{N})}$$

This allows us:

- To find the highest amplification ratio in the system's state-space.
- To find the output matrix \mathbf{C}_{\max} which corresponds to this "worse-case scenario"; we find it as the vector in the SVD decomposition of \mathbf{N}) matrix corresponding to the largest singular value.

MIMO FREQUENCY RESPONSE, 4

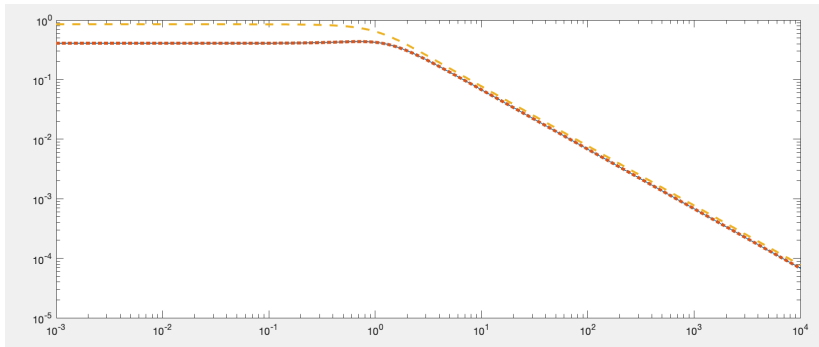
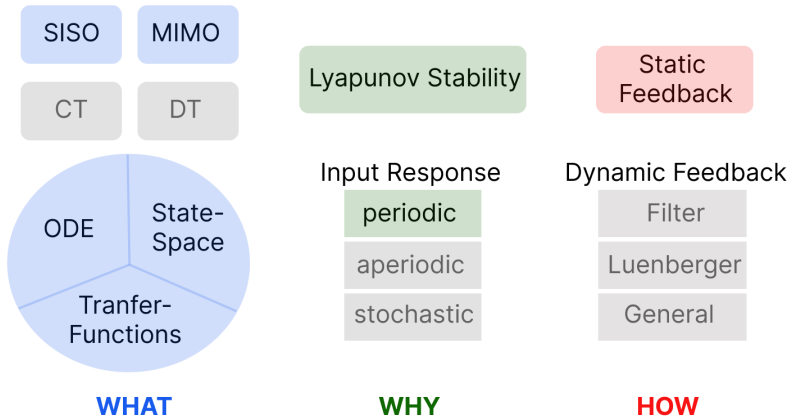


Figure 1: Dashed - worst case amplitude response, other two - a particular mode plot

WHERE ARE WE



Lecture slides are available via Github:

github.com/SergeiSa/Control-Theory-2025

