

# Final Report: Pitch-Rate Tracking CAS of Exam 2 Airplane to Meet Certain Specifications

## Step 1: Develop a Short Period Approximation

First and foremost, our full state space model of our exam 2 airplane can be shown by the following:

$$\dot{x} = A \cdot x + B \cdot u$$

$$y = C \cdot x + D \cdot u$$

u is input. columns: [thrust, elevator]

x contains the states [velocity; alpha (angle of attack); theta (pitch); q (pitch-rate)]

$\dot{x}$  contains the derivative of the states

y contains the output

A,B,C,and D are matrices as shown below:

```
A_full = 4x4
-0.0118  18.9953 -32.1700    0
-0.0003  -1.2608    0    1.0000
    0      0      0    1.0000
 0.0000  -3.1046    0   -1.0595
```

```
B_full = 4x2
 8.1996    0
-0.0002    0
    0      0
 0.0200  -0.0440
```

```
C_full = 4x4
 1.0000    0    0    0
    0   57.2958    0    0
    0    0   57.2958    0
    0    0    0   57.2958
```

```
D_full = 4x2
 0    0
 0    0
 0    0
 0    0
```

Because alpha and pitch rate are the main contributors to the short period mode of the aircraft, they can be extracted from the full model matrices to make a short period approximation:

```
A_sp = 2x2
-1.2608    1.0000
-3.1046   -1.0595
```

```
B_sp = 2x1
 0
-0.0440
```

```
C_sp = 2x2
 57.2958    0
    0   57.2958
```

```
D_sp = 2x1
 0
```

0

We can ensure this is accurate by seeing how the eigenvalues of the short period A matrix are nearly identical to short period eigen values of the full model A matrix:

```
A_full_eig = 4x1 complex
-1.1612 + 1.7594i
-1.1612 - 1.7594i
-0.0049 + 0.0785i
-0.0049 - 0.0785i

A_sp_eig = 2x1 complex
-1.1602 + 1.7591i
-1.1602 - 1.7591i
```

## Step 2: Determine Alpha Gain, Compensator Zero, and Pitch-Rate Gain to meet the Short Period Specs

### Include Additional Hardware

First and foremost, additional hardware was included. The entire system of the aircraft is not just the plant i.e. our original matrices. We must incorporate the hardware for

1. an angle-of-attack (AoA) sensor, along with a filter with transfer function  $10/(s+10)$
2. a pure pitch-rate sensor
3. actuator motor with transfer function  $20.2/(s+20.2)$

By making the state space of these transfer functions and then using 'series()' command in matlab, we can incorporate these pieces of hardware into the existing plant state space to create a new state space.

We now have new state space matrices that represent actuator + plant + alpha filter:

```
AA = 4x4
-10.0000  572.9578         0         0
         0   -1.2608     1.0000         0
         0   -3.1046    -1.0595     0.0440
         0         0         0    -20.2000

BB = 4x1
         0
         0
         0
    20.2000

CC = 2x4
    1.0000         0         0         0
         0         0    57.2958         0

DD = 2x1
         0
         0
```

### Implement Gains

Now, the alpha gain ( $k_a$ ), compensator zero i.e. integrator gain ( $k_i$ ), and pitch rate gain ( $k_p$ ) must be incorporated.

For initial values of  $k_a$ ,  $k_i$ , and  $k_p$ , arbitrary guesses were made, in this case 0.02, 3, and 0.5 respectively.

The first one to focus on would be  $k_a$ . We can implement it by closing the alpha loop of our control system.

This is done by multiplying our 'CC' matrix by our  $k_a$  vector  $[k_a \ 0]$  ( $k_a$  only affects alpha, not  $q$ ), and then by our 'BB' matrix. This result is then subtracted from our previous matrix 'AA' to create a new A matrix.

We now have a new state space. The only change is to the 'AA' matrix, now 'AAc1':

```
AAc1 = 4x4
-10.0000  572.9578      0      0
      0   -1.2608    1.0000      0
      0   -3.1046   -1.0595    0.0440
-21.4181      0      0   -20.2000
```

Next,  $k_i$  can be implemented by combining the integrator state space (its B matrix contains our  $k_i$  value) with the state space we just got after implementing  $k_a$ .

The integrator state space:

```
Integrator =
```

```
A =
      x1
x1    0
```

```
B =
      u1
x1  3.566
```

```
C =
      x1
y1    1
```

```
D =
      u1
y1    1
```

```
Continuous-time state-space model.
```

New state space after combining the integrator state space with our previous state space:

```
AAA = 5x5
-10.0000  572.9578      0      0      0
      0   -1.2608    1.0000      0      0
      0   -3.1046   -1.0595    0.0440    0
-21.4181      0      0   -20.2000  20.2000
      0      0      0      0      0
BBB = 5x1
      0
      0
      0
 20.2000
  3.5662
CCC = 2x5
  1.0000      0      0      0      0
      0      0  57.2958      0      0
DDD = 2x1
      0
      0
```

Finally,  $k_p$  can be implemented in a similar way as  $k_a$  was. However, only the sole value of  $k_p$  is needed as we will multiply it only by the second row of 'CCC' which represents pitch rate. Then again, multiplied by our current B matrix and subtracted from our current A matrix makes a new A matrix.

This is our final state space that relates input to pitch rate and alpha (first row of C matrix is for alpha, second row is for pitch rate):

```

aq_r_ss_A = 5x5
103 x
    -0.0100    0.5730    0    0    0
         0   -0.0013    0.0010    0    0
         0   -0.0031   -0.0011    0.0000    0
    -0.0214    0   -2.1321   -0.0202    0.0202
         0    0   -0.3764    0    0
aq_r_ss_B = 5x1
    0
    0
    0
    37.2124
    6.5697
aq_r_ss_C = 2x5
    1.0000    0    0    0    0
         0    0   57.2958    0    0
aq_r_ss_D = 2x1
    0
    0
q_r_tf =

          93.88 s^3 + 1392 s^2 + 4954 s + 4221
-----
s^5 + 32.52 s^4 + 370.4 s^3 + 1995 s^2 + 6391 s + 4221
Continuous-time transfer function.

a_r_tf =

          938.8 s + 3348
-----
s^5 + 32.52 s^4 + 370.4 s^3 + 1995 s^2 + 6391 s + 4221
Continuous-time transfer function.

```

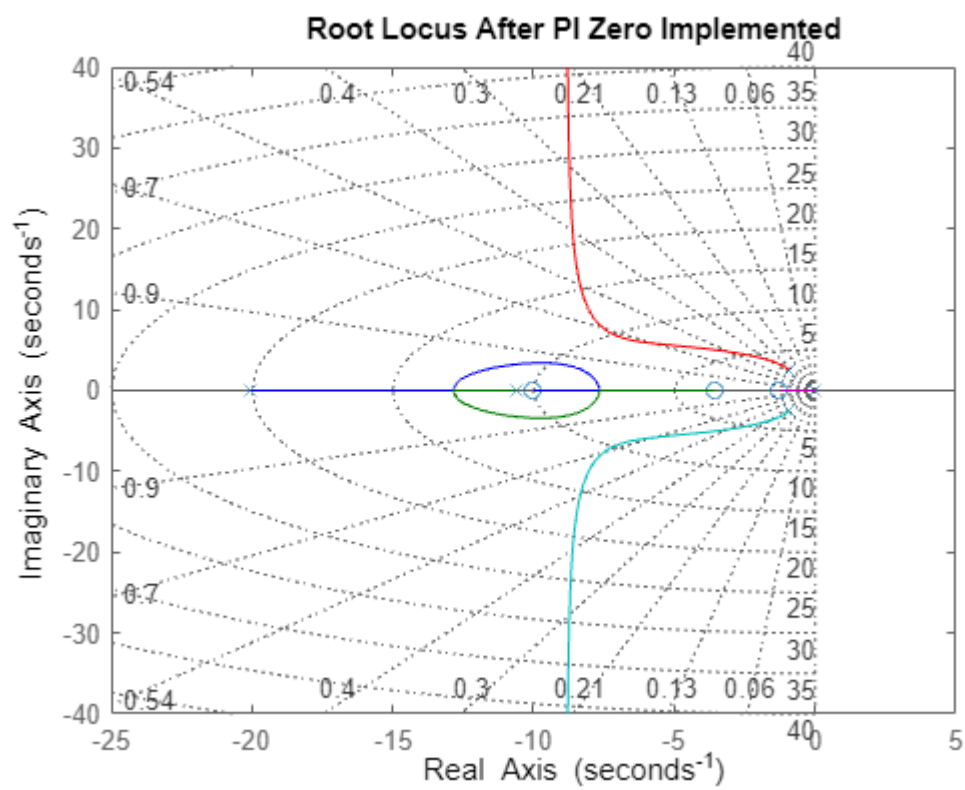
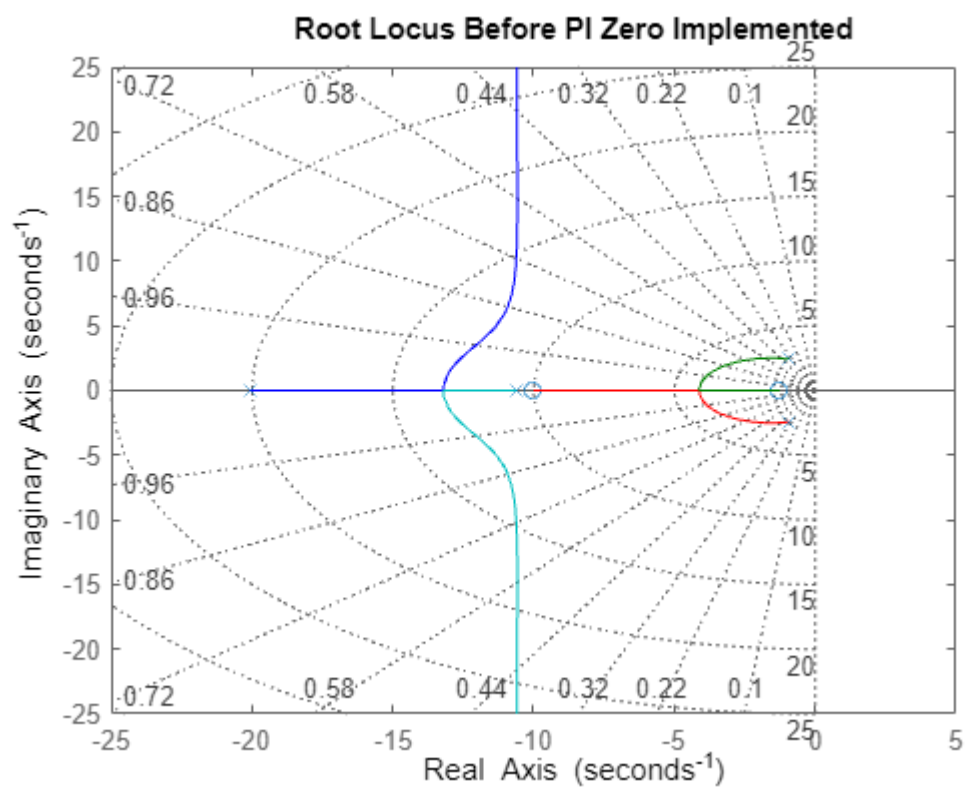
## Root Locus

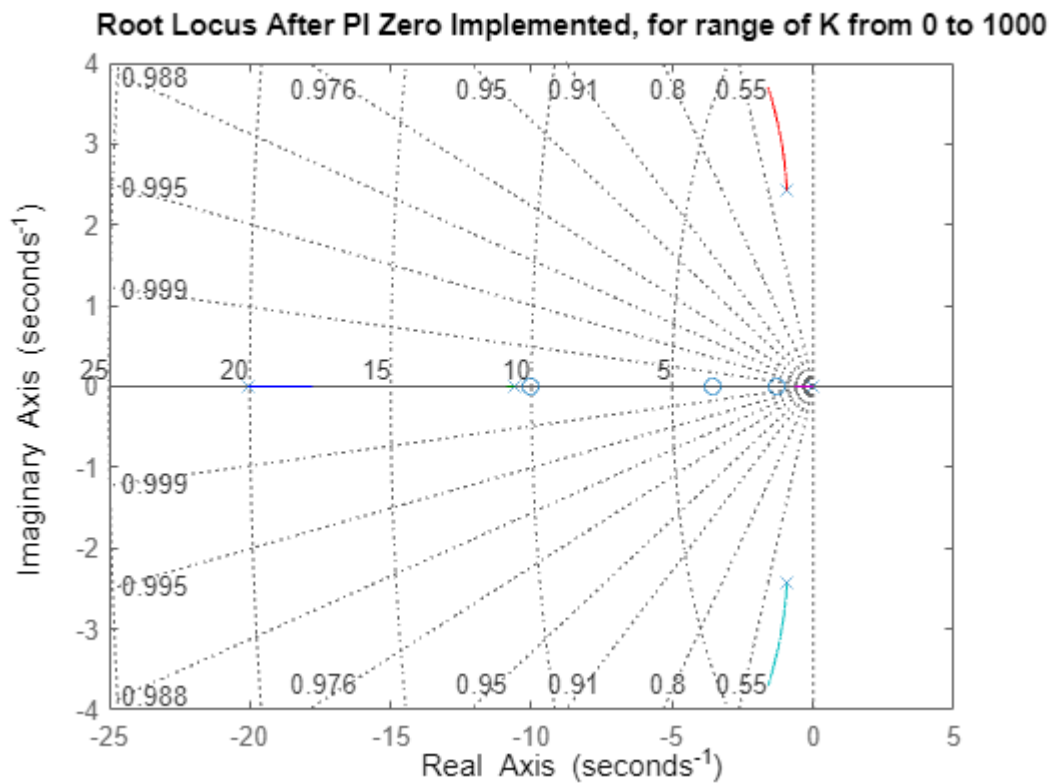
The root locus before and after the PI zero was implemented can be seen below

By adding the zero, we can ensure that the lowest  $k_p$  values will create the lowest oscillatory frequency system.

Additionally, it also drastically decreases the oscillation growth rate as  $k_p$  increases.

Lastly, it creates a general trend as  $k_p$  increases that reduces the damping ratio, meaning that low values of  $k_p$  will be appropriate for our design.





### Frequency, Damping Ratio

Next we calculated the Frequency and Damping Ratio of our short period approximation system.

The frequency is the largest imaginary part of our eigen values of our final state space A matrix

The natural frequency is the square root of the frequency squared, and its corresponding real part squared.

i.e.  $\text{freq\_nat} = \sqrt{\text{corresponding\_real}^2 + \text{largest\_imag}^2}$

i.e. magnitude of the distance from the origin to the pole.

The damping ratio is the negative of the corresponding real part divided by the natural frequency.

By doing this, we got values of:

```
freq = 4.6343
```

```
damping_ratio = 0.5139
```

We attempted to achieve the suggested values of a frequency of 5.80 and damping ratio of 0.70.

We did this by placing all our previous work in a function and adding the following snippet to the end which shows how far we are from the suggested values.

```
frequency_err = (frequency - 5.80)^2;
```

```
damping_ratio_err = (damping_ratio - 0.70)^2;
```

```
err = freq_err + damping_ratio_err;
```

By passing our gain parameters into this function and returning the error, this mimics linear regression to solve for the lowest error in our function. MATLAB's 'fminsearch()' can be used on this function to recursively find the lowest error, i.e. values of our 3 gains that bring the frequency and damping ratio as close to 5.8 and 0.70 as possible.

They were found as follows:

Optimal ka: 1.060294

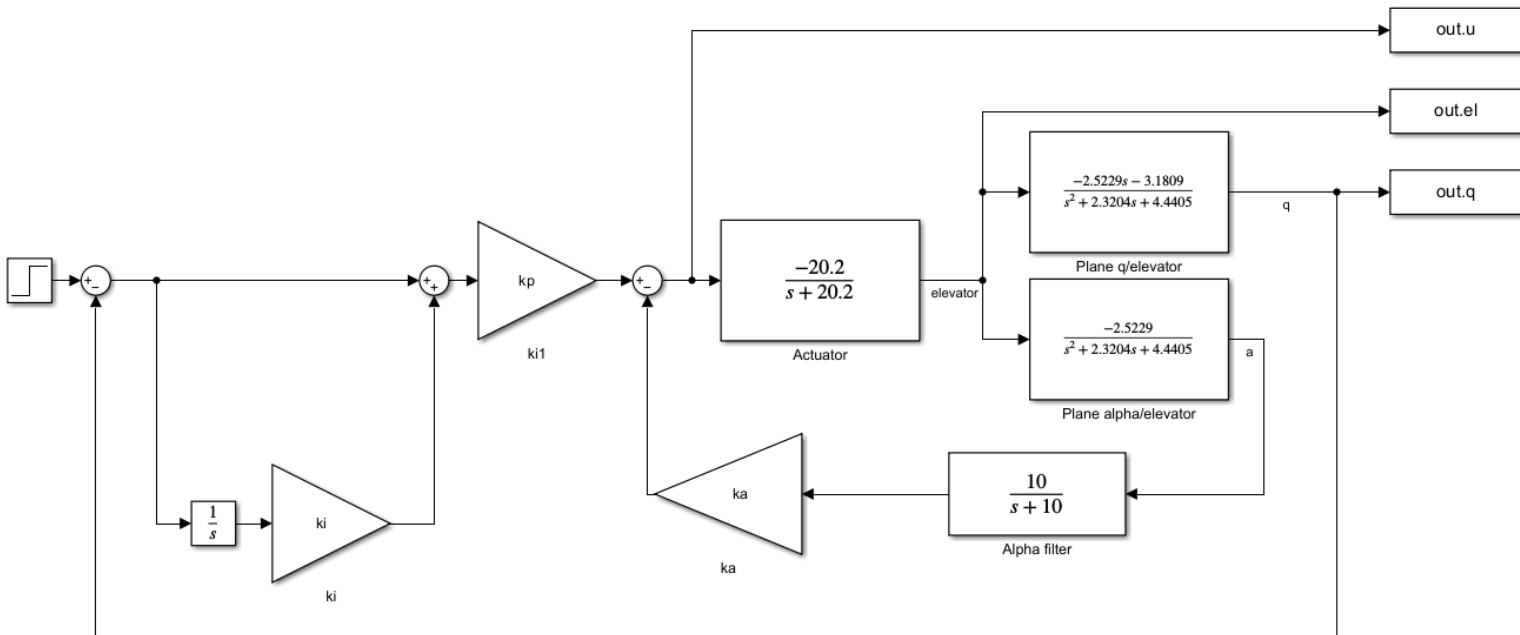
Optimal ki: 3.566154

Optimal kp: 3.195926 (later adjusted to 1.8422 manually, see note below)

Note: When using these values in our Simulink simulation in the next section, elevator response surpassed the 2 degrees max, so kp was manually adjusted to correct for this.

### Step 3: Plotting Response With Simulink

A block diagram of the entire short period approximatn system was created.

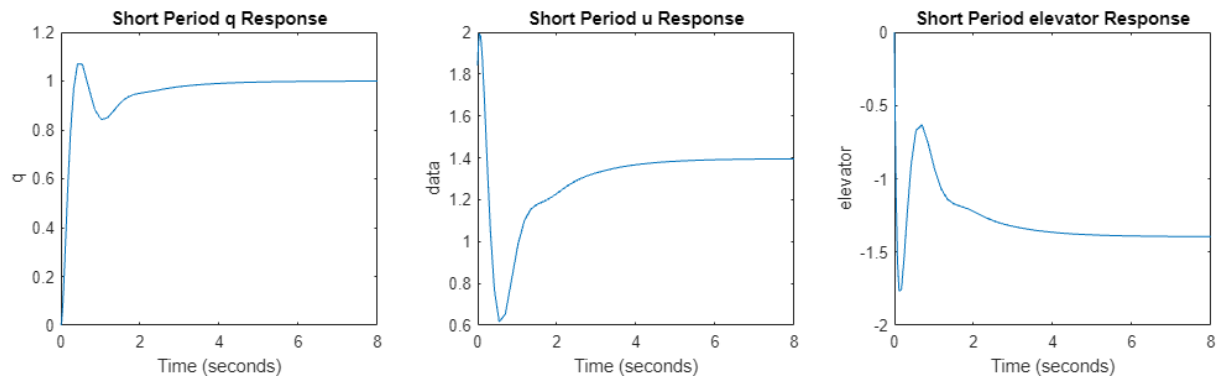


In order to simply the simulink model, we decided to convert our state spaces from our matlab code into transfer functions (using 'ss2tf()'). This required 2 transfer functions so that we can have an output of alpha and q.

The pitch rate and elevator response can be easily plotted now because in simulink, the signal can be easily pulled from any point in the system.

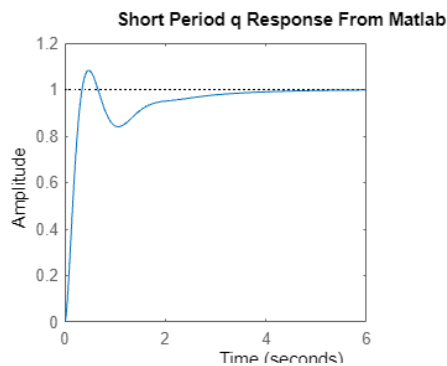
This can be seen in the block diagram above where the u signal that goes to our motor, and elevator response after the actuator motor, are both collected as output.

A matlab script was written to run the simulation and plot the output of the responses for easy modification of the control parameters. After kp was manually adjusted to bring the maximum allowable elevator deflection (from its equilibrium value) down to below 2 degrees, the following plots were generated:



We decided to record the signal before the actuator as well to show that the  $u$  input to the plant is not actually the elevator and the sign change in our actuator transfer function is necessary to ensure the elevator responds in the correct direction.

Note: The  $q$  response plot can still be generated from the matlab code before using Simulink. That plot can be seen below and from that plot we can confirm the settling time is less than 5 seconds:

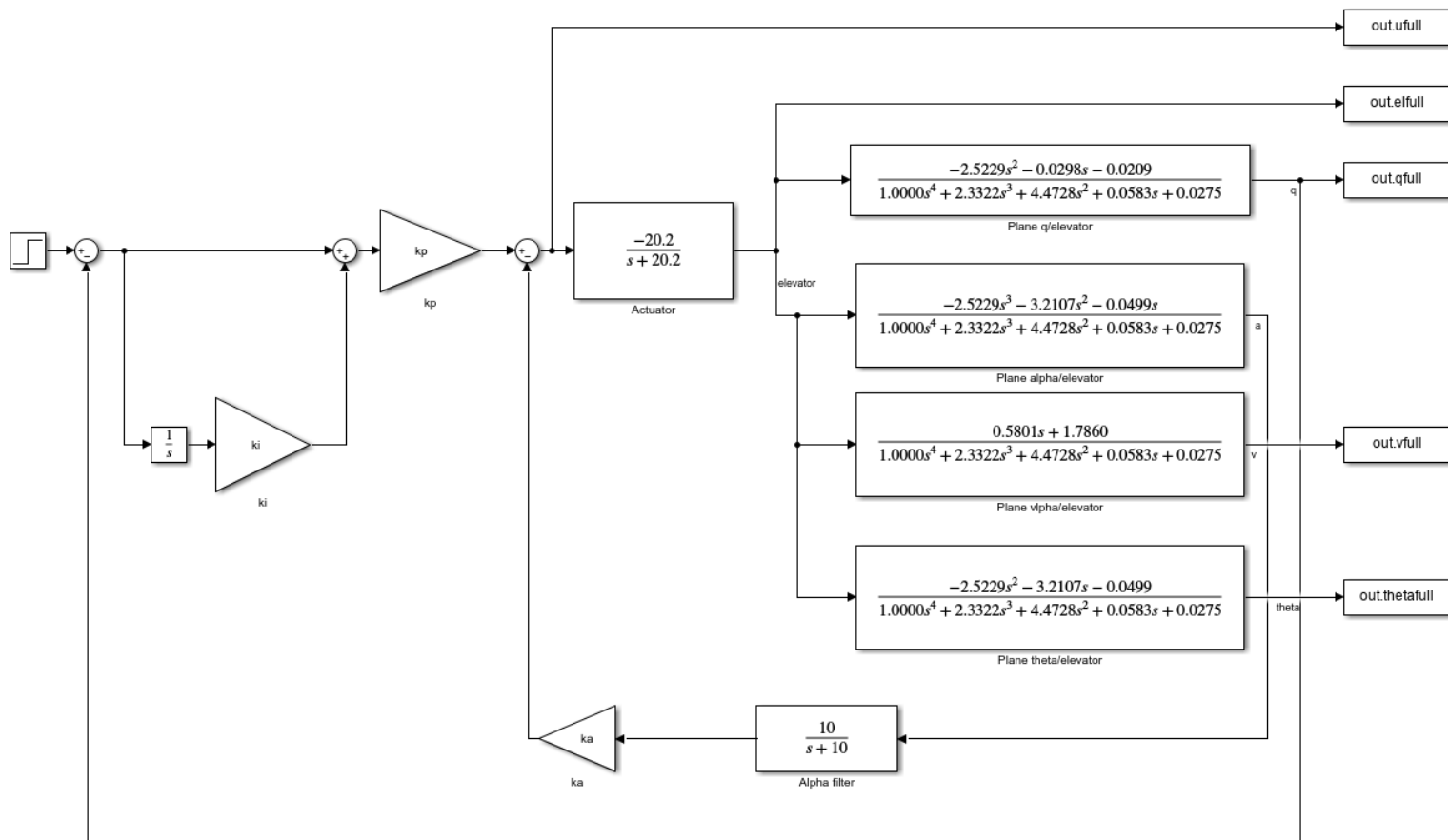


```
settling_time = 3.1477
```

## Step 4: Full Model

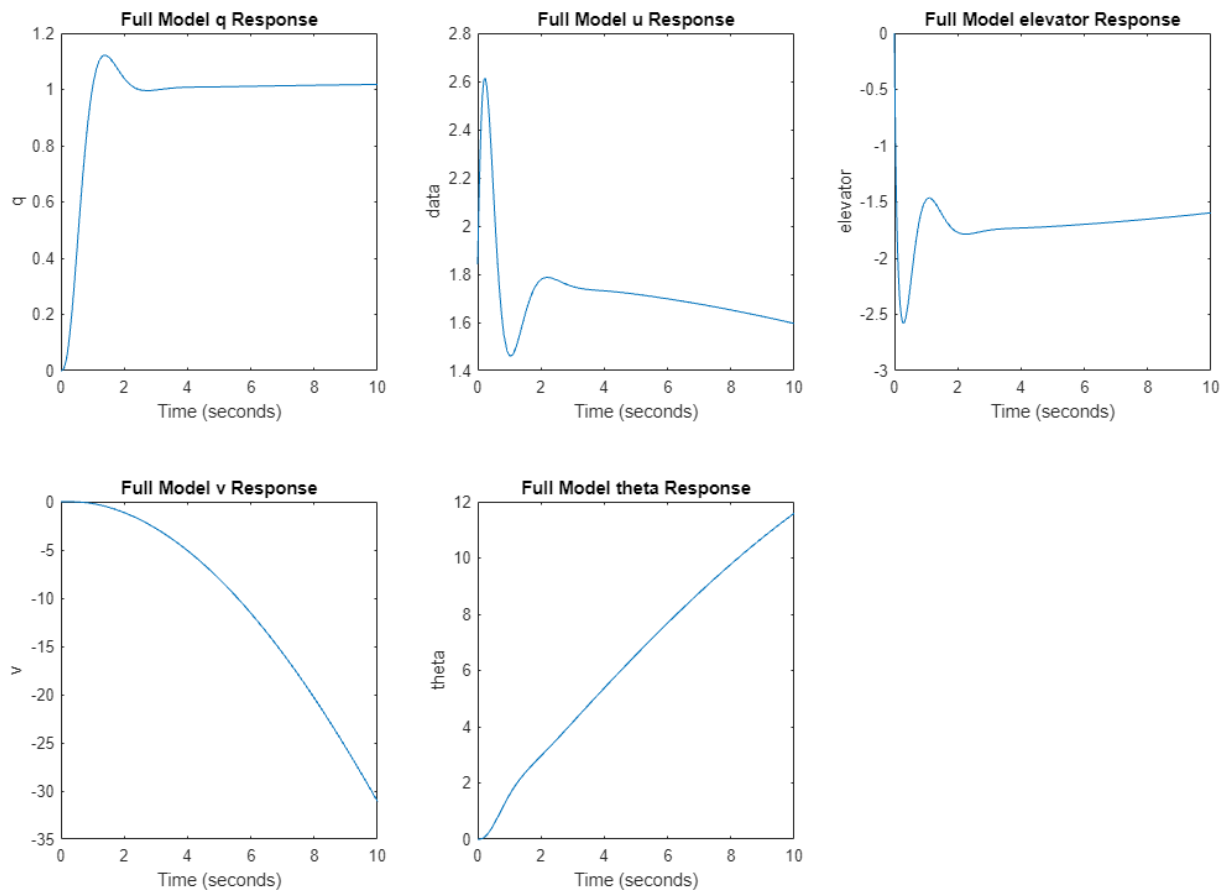
A block diagram of the entire full model of the system was created.





As we were using transfer function and not state space models in simulink, 2 more transfer functions needed to be added to output the 2 additional states, the velocity and pitch of the airplane.

Below are plots for all state responses for the full model system with simulations for both 10 seconds duration and 1000:



Comparing the elevator response from the 10 second plot of the full model to the short period approximation, it can be seen that the overshoot passes 2 degrees in the full model while it does not for the approximate model, where it is just under 2 degrees. With this information, we can conclude that the approximation may not always be accurate enough to create a control system for the full model, especially if you design the controller to the limit of the specifications. To adjust for this, we can add a buffer factor when using the approximate model to design the control values to ensure the full model response is appropriate.

Additionally, it can be seen that the frequency and settling time of the q response from the full model differ slightly from the short period approximate model. In this case the frequency is lower, and the settling time is smaller. When designing a system using approximations, all of this needs to be considered, which is why it is important to verify our design with the full model after the values are chosen.

