


As a future member of the engineering profession, the student is responsible for performing the required work in an honest manner, without plagiarism and cheating. Submitting this work with my name and student number is a statement and understanding that this work is my own and adheres to the Academic Integrity Policy of McMaster University and the Code of Conduct of the Professional Engineers of Ontario. Submitted by [Shalmi Patel, patels19, 400023762]



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Technical Report

1C03
Dr. Doyle

Group 41

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Team Contributions and Meetings

Team Contributions	
Member Names	Contributions
Ali Shah	Content and Report Compilation
Kylee Schram	Calculations and Solid Models
Shalmi Patel	MapleSIM and Drawings
All members	Physical model assembly

Team Meetings		
Meeting Number	Members Present	Meeting Objective
1	Ali, Kylee, Shalmi	Project Outline
2	Kylee, Shalmi	Solid Models
3	Ali, Kylee	Milestone I
4	Kylee, Shalmi	Signatures
5	Ali, Kylee	Milestone II
6	Kylee, Shalmi	MapleSIM
7	Ali, Kylee	Technical Report
8	Ali, Kylee, Shalmi	Conclusion

Introduction

We have been hired by company XYZ Mechanisms and tasked with the modification of a CD player's power train design. Previously used components are now obsolete, thus a design change is necessary. The player's motor had a set operating speed, 8610 rpm, which needed to provide a linear output velocity of 0.105m/s at the read-head. Using gear ratio equations, specifications were calculated. The final assembly uses a minimal number of gears to transmit the required velocity to the read-head while remaining within the space constraints of the chassis.

As a secondary requirement, a scaled-up, 3D-printed model of the mechanism was produced. A mounting bracket was also fabricated to compliment the prototype.

Mechanism Overview

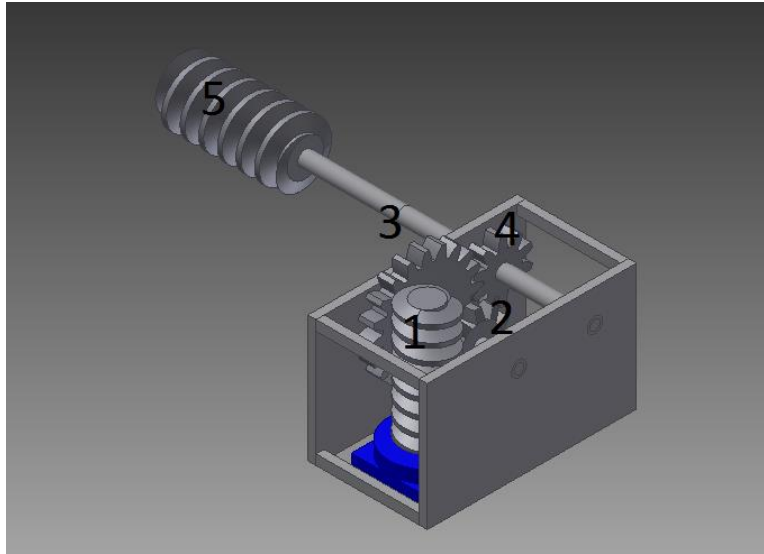


Figure:1

The mechanism relies on a worm drive, [1, 2] to reduce the output speed of the motor. The worm [1] is driven at the motor's operating speed, and the worm gear [2] decreases this speed by a factor equal to its number of teeth. The worm gear [2] is axially connected to spur gear [3], and spur is driven at the cut speed. Spur gear [3] is meshed to spur [4]. Spur [4], with half the number of teeth of [3], doubles the speed, and drives the final worm [5] at this angular velocity. As the final worm [5] rotates, the read head moves in a linear direction along its length.

Material Volume Summary

Mechanism volume: 6242.669 mm³

Volume with mounting: 9902.753 mm³

Gear [refer to figure 1]	Volume (mm ³)
Worm 1 [1]	1217.008 mm ³
Worm gear [2]	194.689 mm ³
Spur Gear 1 [3]	913.720 mm ³
Spur Gear 2 [4]	214.672 mm ³
Final worm gear [5]	1970.052 mm ³

Calculations

Gear	Diameter (mm)	Module	# of Teeth	W (rev/s)	ACP	Linear Velocity (mm/s)
Worm 1	10	1	1	143.5	3.14	N/A
Worm Gear	10	1	10	14.35	N/A	N/A
Spur 1	20	1	20	14.35	N/A	N/A
Spur 2	10	1	10	28.7	N/A	N/A
Worm 2	12.34	$3.66/\pi$	1	28.7	3.66	105.04

Target Output: 105 mm/s

$$\text{Input } 210 \times 41 = 8610 \text{ rev/min} \\ = 143.5 \text{ rev/s}$$

- Final rotational output
 $\frac{105 \text{ mm/s}}{\text{ACP}} \rightarrow \text{ACP constrained but 1 and 4}$
- ACP = 3.67

$$\text{So, } \frac{105 \text{ mm/s}}{3.67 \text{ mm/rev}} = 28.61 \text{ rev/s}$$

Stage One: Reduction

Worm Drive A.1.

• Diameter 6 mm

• $\omega = 143.5 \text{ rev/s}$

Worm Gear A.2.

$$\frac{\omega_1}{\omega_2} = \frac{z_2}{z_1} \quad \text{target } \omega_2 \text{ is } \frac{\omega_1}{10} = 14.35 \text{ rev/s}$$

$$\frac{143.5}{14.35} = \frac{z_2}{1} \quad \text{Module 1} \\ \text{diameter 10} \\ \text{teeth 10}$$

$$10 = z_2$$

Spur Gear B

• Axial mount to A.2.

• diameter 20 to facilitate increase in next stage, so

• 20 teeth

• module 1

• 14.35 rev/s

Spur gear C
 • mesh to B
 Diameter = $\frac{D_p}{2}$
 $= \frac{20}{2}$
 $= 10$
 teeth 10, module 1

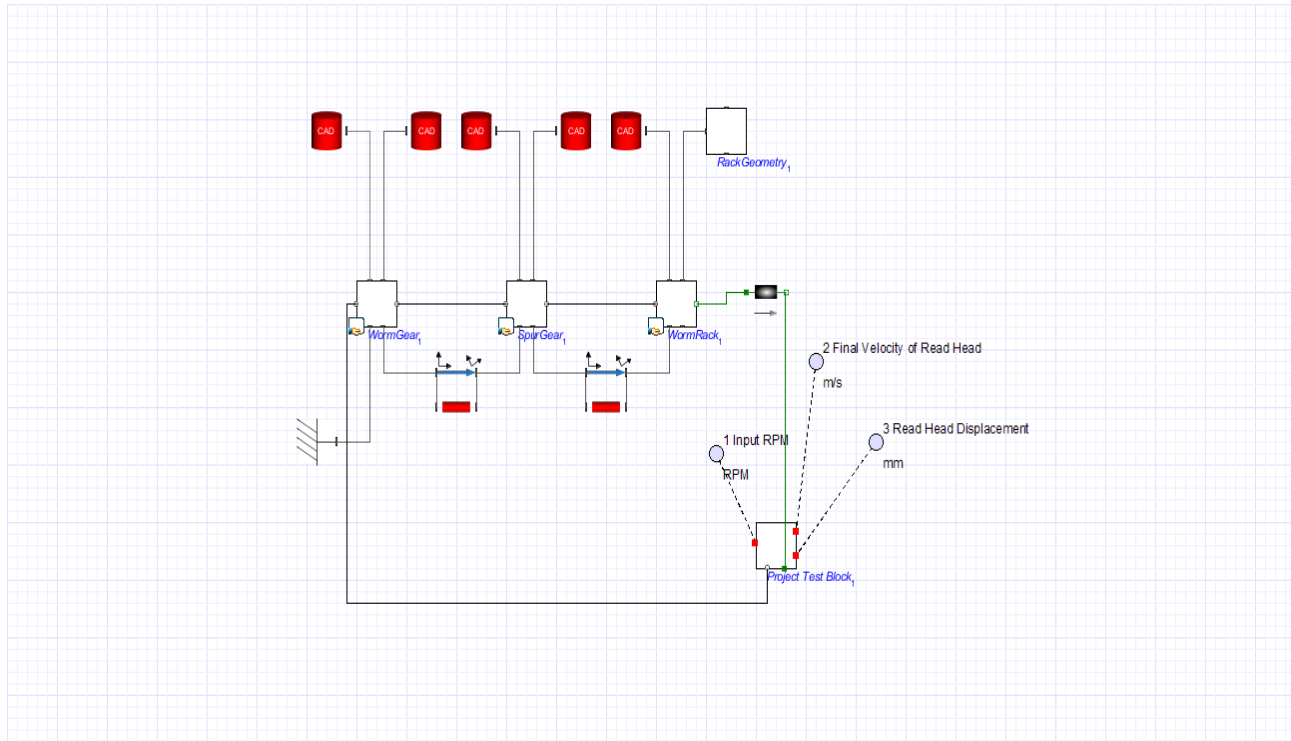
Spur gear D
 • mesh to C
 • maintain w , make distance
 so $w_2 = w_1$, $D_2 = D_1$
 diameter = 10
 teeth = 10
 module = 1

Final worm
 ACP 3.67 $\frac{V}{ACP} = w$
 $v = 105$
 $v = (3.67)(28.7)$
 $\approx 105.32 \text{ mm/s}$

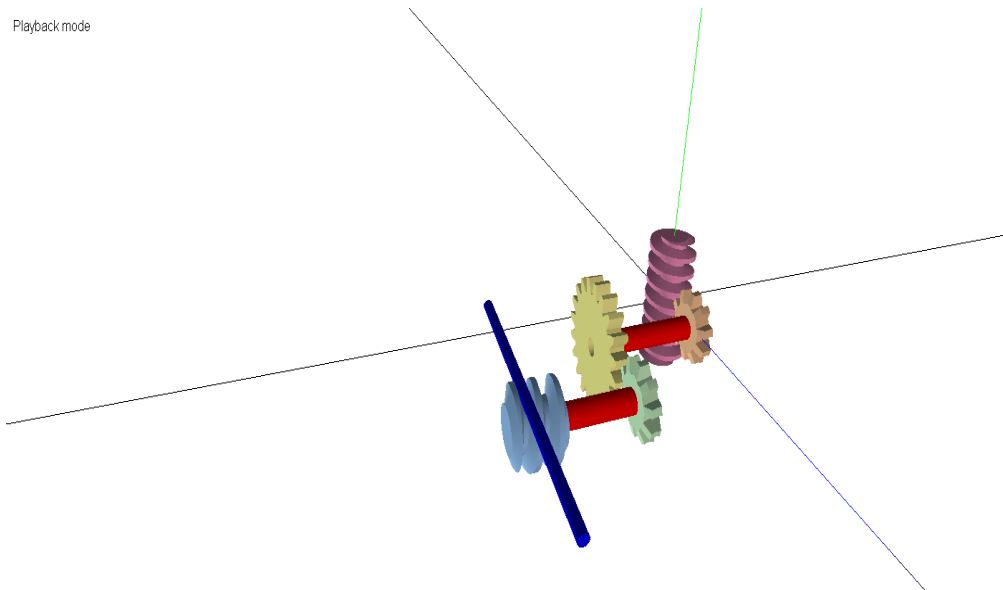
Note: ACP was later modified to 3.66, to give a predicted linear velocity of 105.042mm/s. This was the value used for simulation and physical model. "Spur Gear D" was omitted.

Simulation & Testing

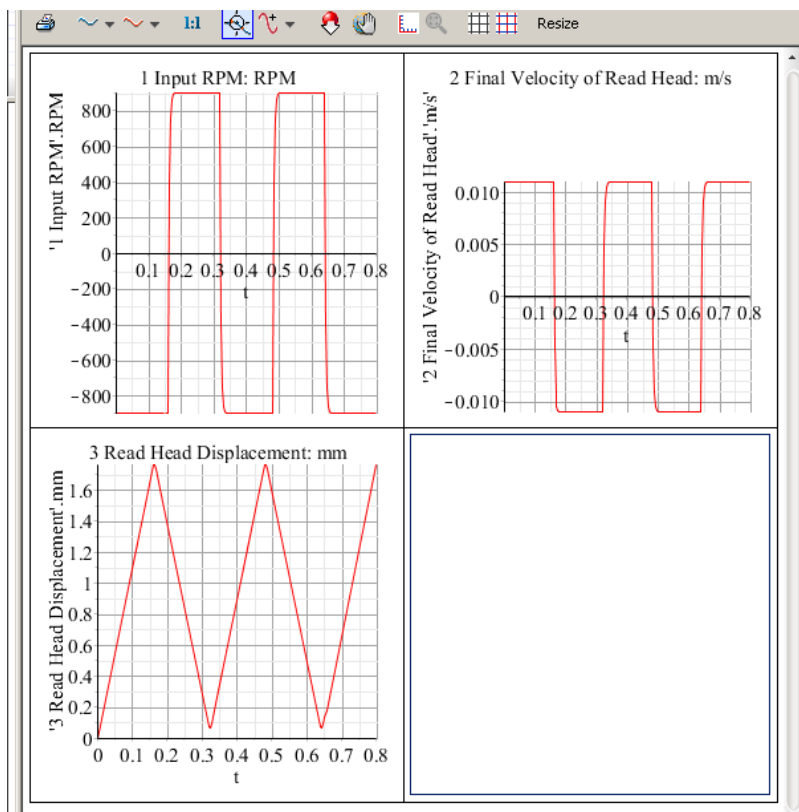
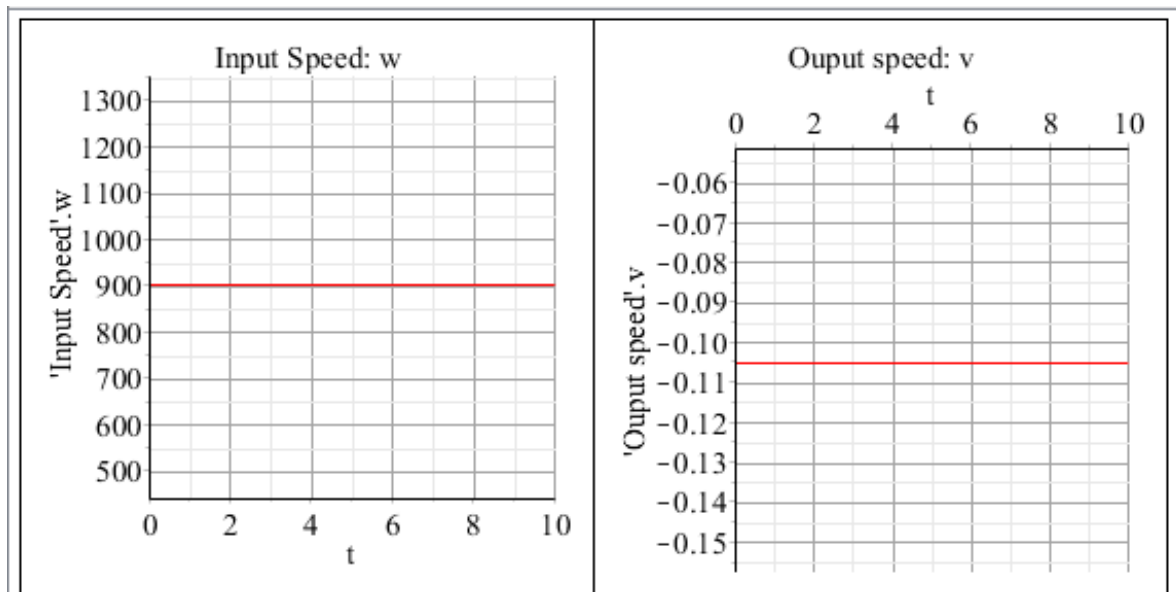
Schematic & Graphical View



Playback mode



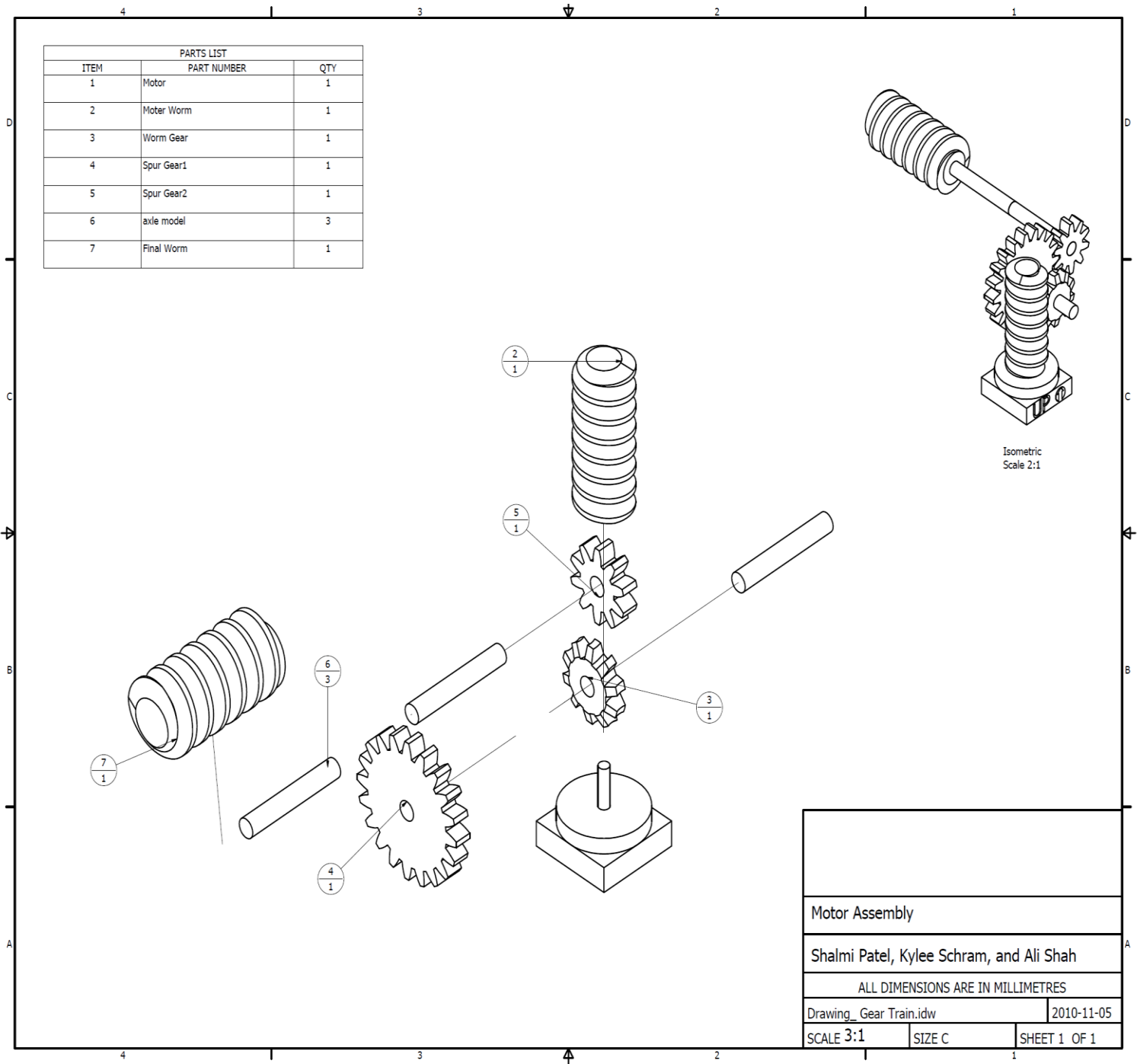
Testing Data



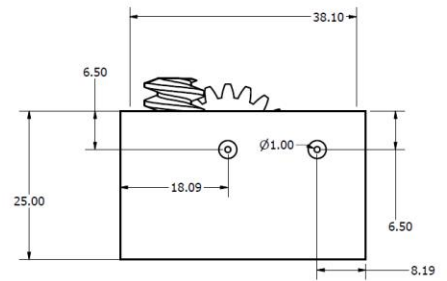
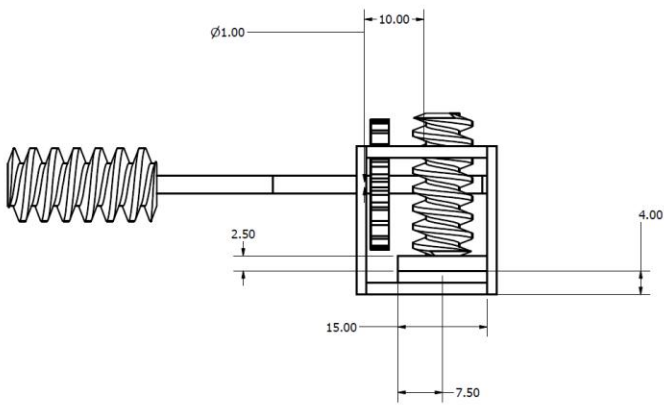
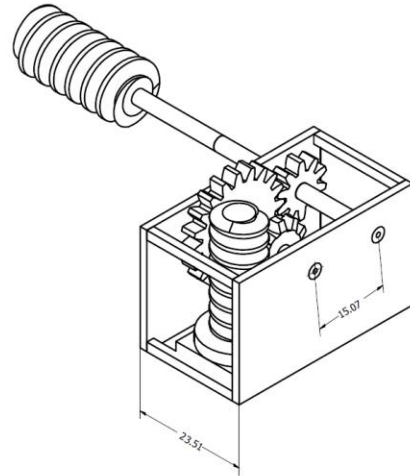
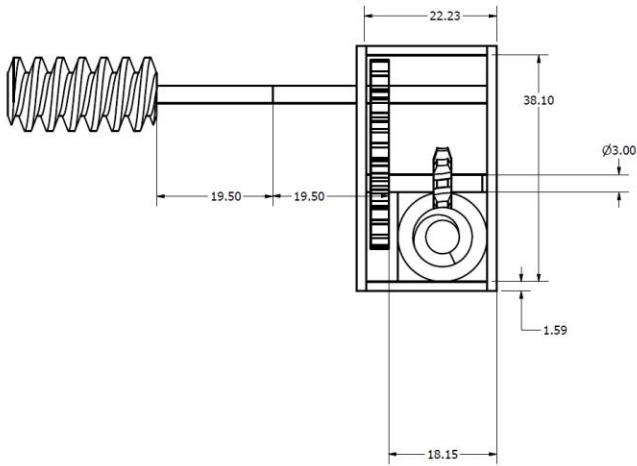
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Drawing Gear Train



Drawing Complete Assembly



Complete Assembly

Shalmi Patel, Kylee Schram, and Ali Shah

ALL DIMENSIONS ARE IN MILLIMETRES

Drawing_Complete Assembly.idw

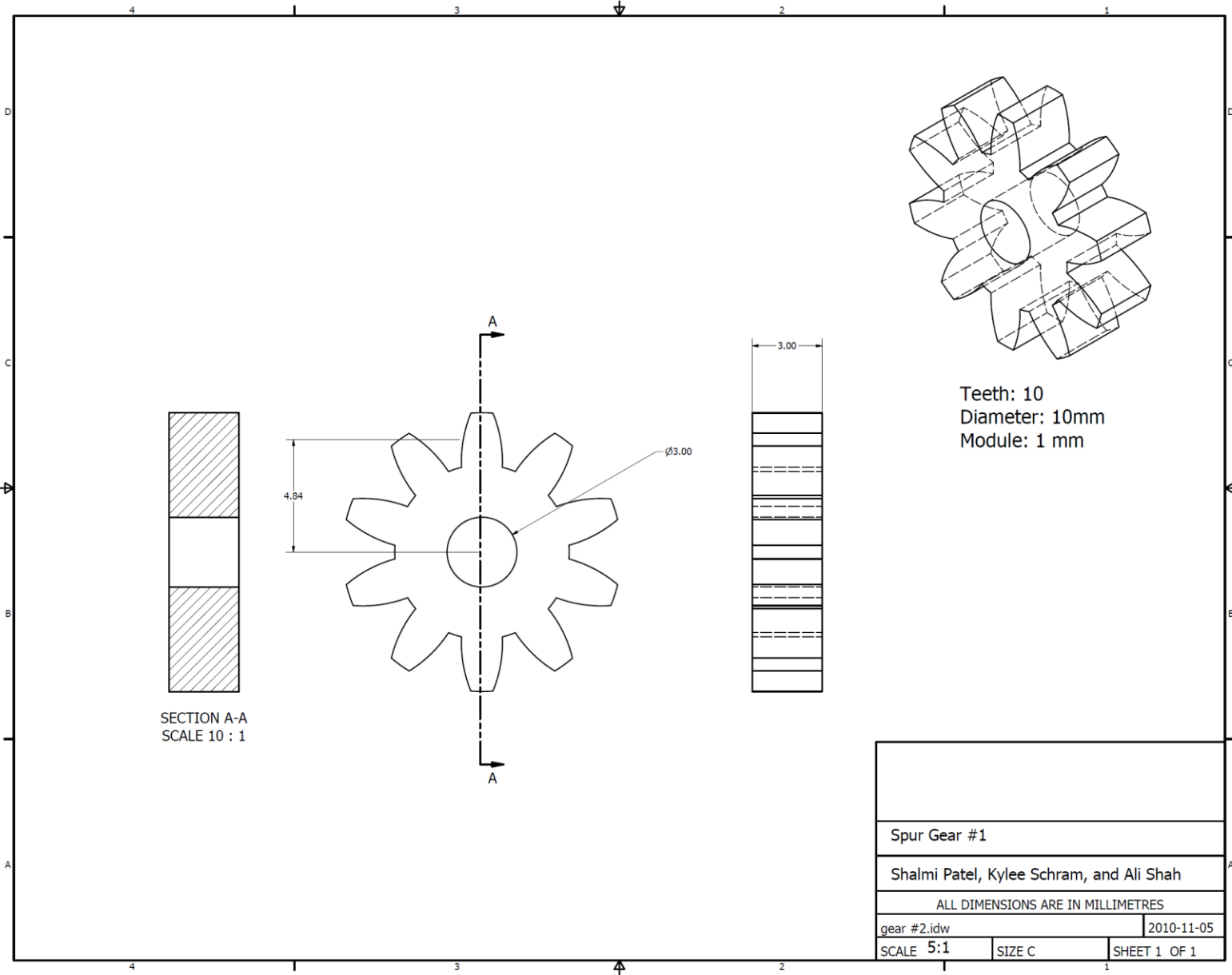
2010-11-05

SCALE 2:1

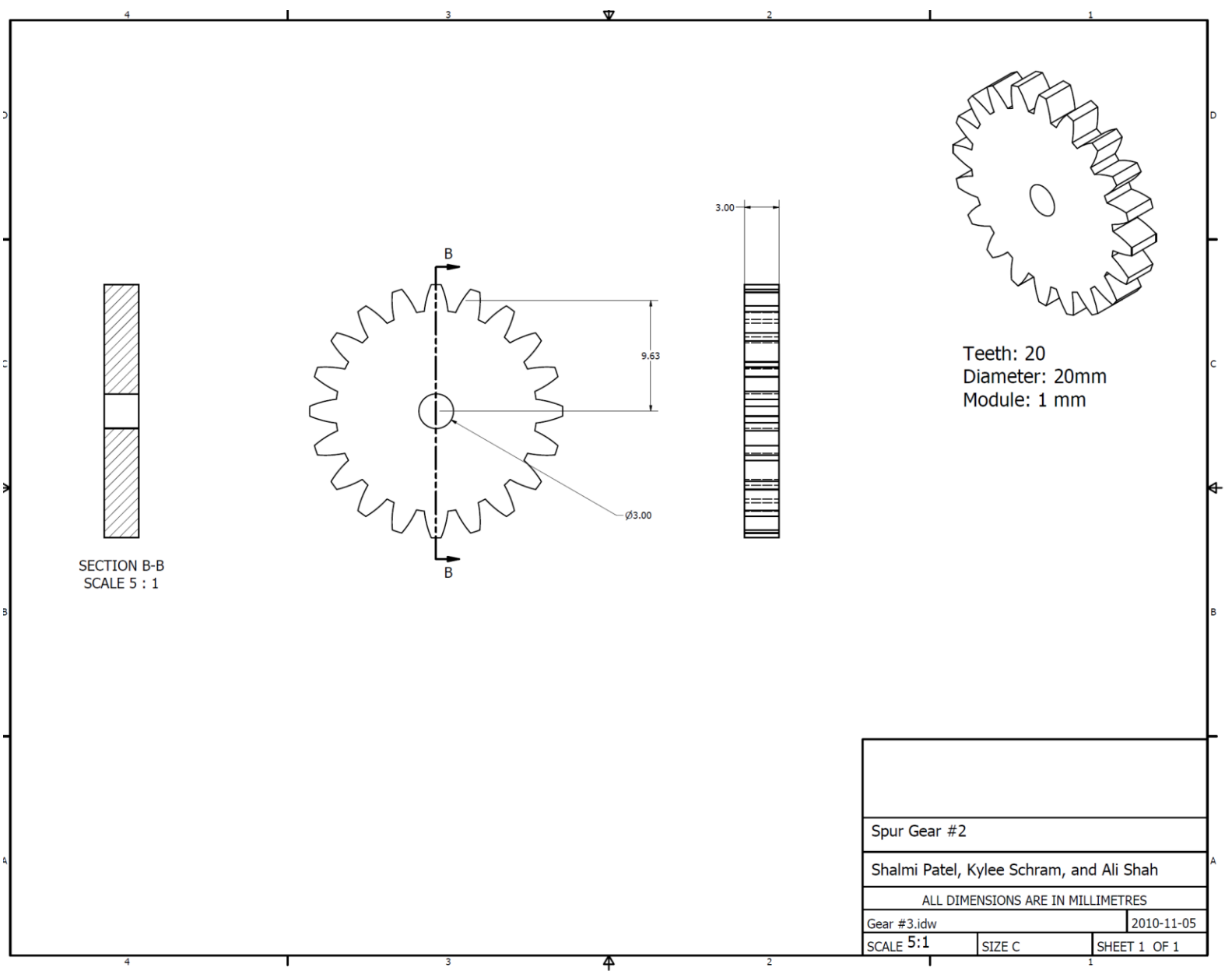
SIZE C

SHEET 1 OF 1

Spur Gear #1

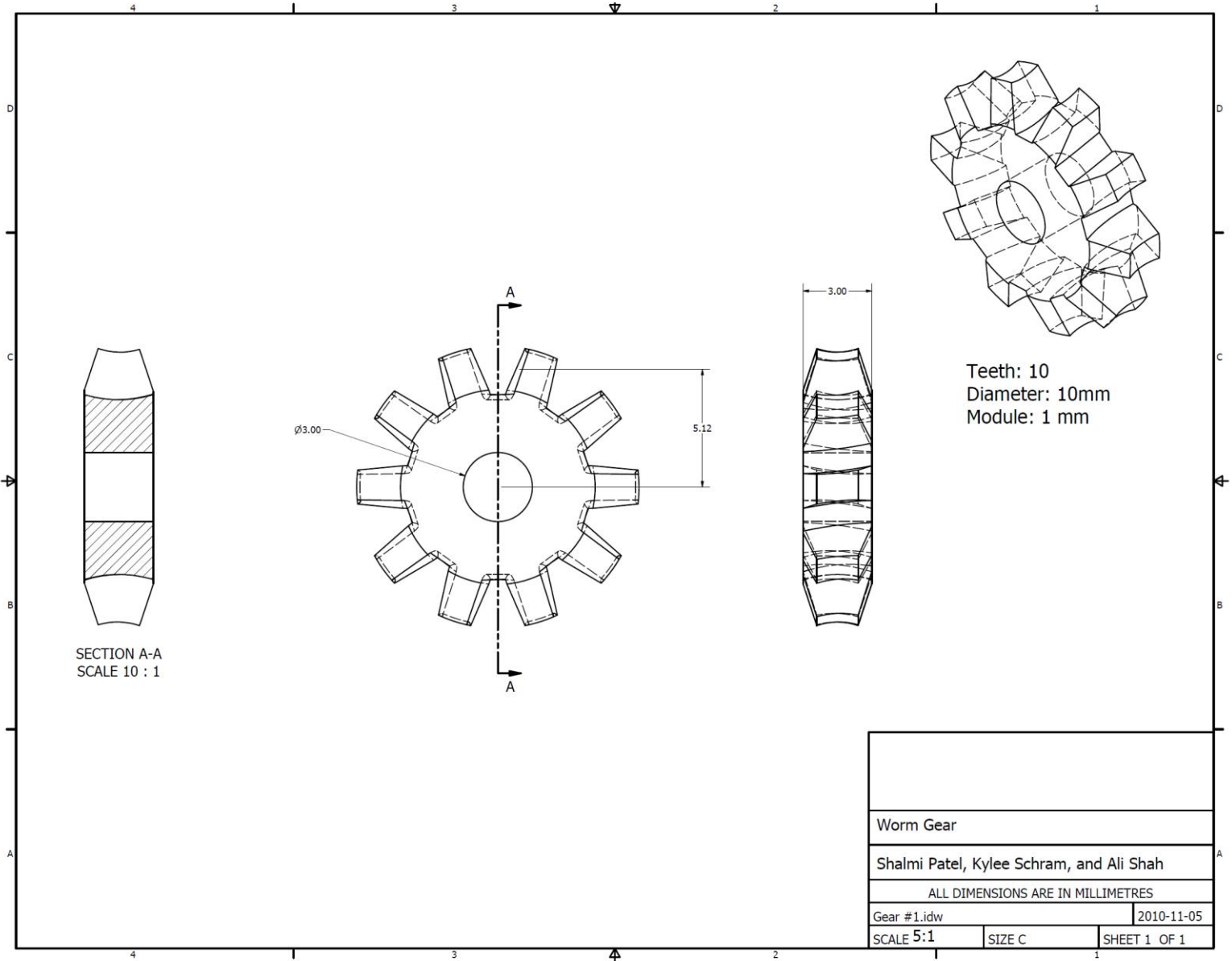


Spur Gear #1

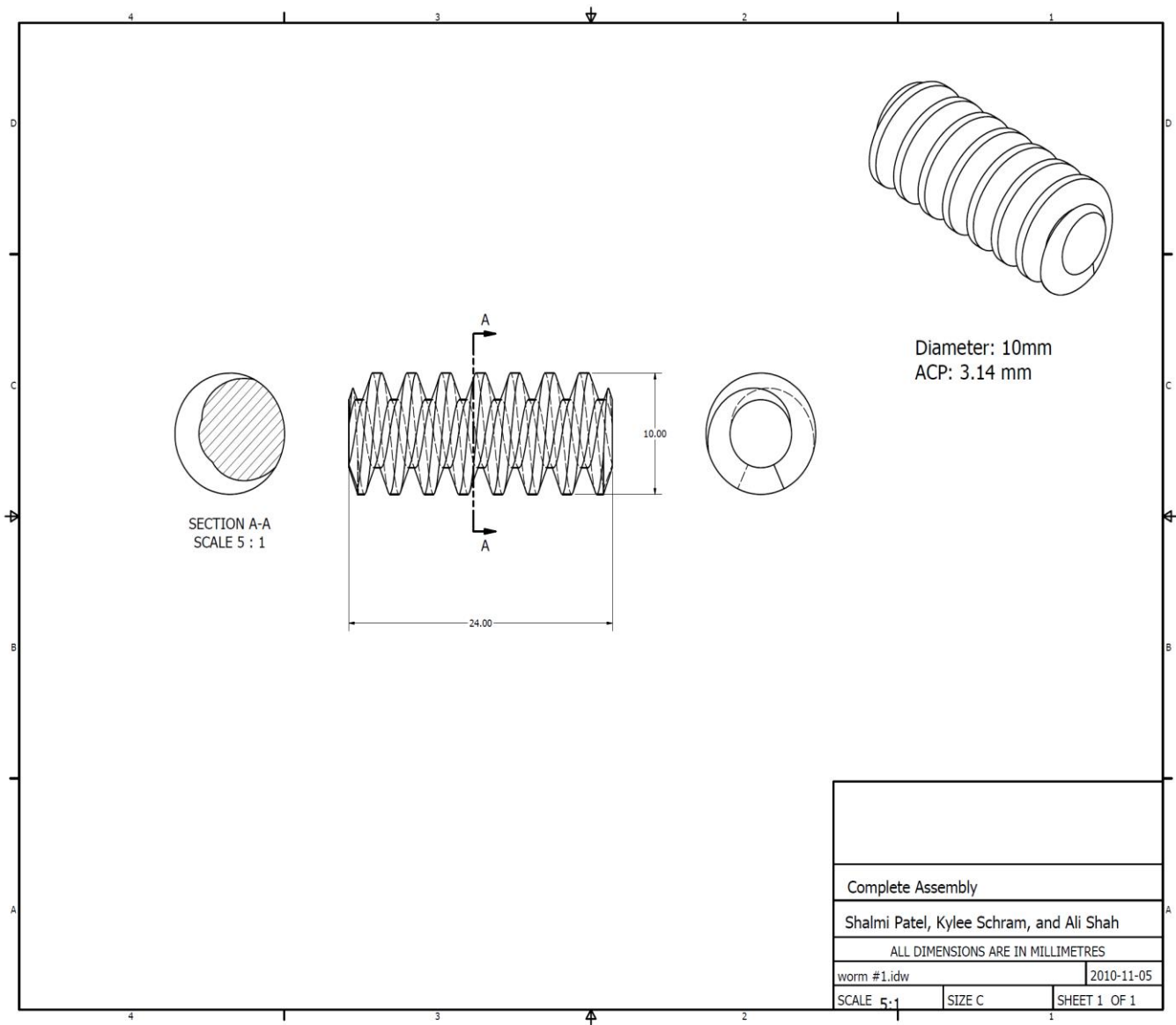


Spur Gear #2		
Shalmi Patel, Kylee Schram, and Ali Shah		
ALL DIMENSIONS ARE IN MILLIMETRES		
Gear #3.idw		2010-11-05
SCALE 5:1	SIZE C	SHEET 1 OF 1

Worm Gear



Worm #1



Worm #2

