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Experiment 10: Effect of P, PI, PD and PID control action on control system

```
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%Batch : Entc EB2

clc;
clear all;
close all;
```

1. Plot the step response for P, PI, PD and PID control systems with unity feedback

(a) For Given System

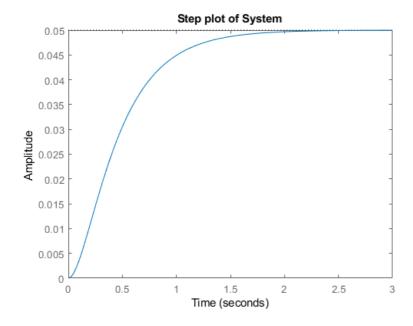
```
N1=1;
D1=[1 10 20];
G1=tf(N1,D1)
figure()
stepplot(G1);
title('Step plot of System')
```

```
G1 =

1

s^2 + 10 s + 20
```

Continuous-time transfer function.

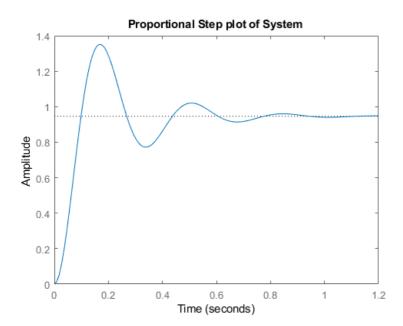


(b) Proportional Control with Kp =350

```
Kp=350;
X1=series(G1,Kp);
System1 = feedback(X1,1)
figure()
```

```
stepplot(System1);
title('Proportional Step plot of System')
```

Continuous-time transfer function.



(c)Integral Control with Ki = 300

```
Ki=300;
N2=Ki;
D2=[1 0];
G2=tf(N2,D2)
X2=parallel(Kp,G2);
X3=series(G1,X2);
System2=feedback(X3,1)
figure()
stepplot(System2);
title('Integral Step plot of System')
```

```
G2 =

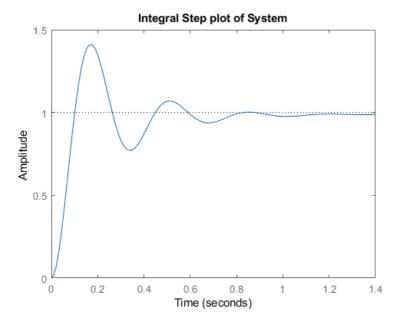
300
---
s

Continuous-time transfer function.

System2 =

350 s + 300
---
s^3 + 10 s^2 + 370 s + 300
```

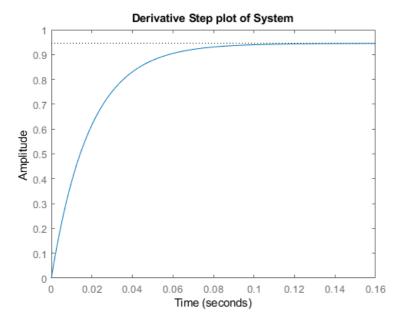
Continuous-time transfer function.



(d) Derivative Control with Kd = 50

```
Kd=50;
N3=[Kd 0]
D3=1
G3=tf(N3,D3)
X4=parallel(Kp,G3);
X5=series(G1,X4);
System3=feedback(X5,1)
figure()
stepplot(System3);
title('Derivative Step plot of System')
```

Continuous-time transfer function.



(e) PID Control with

```
PID=Kp+G3+G2
X6=series(G1,PID);
System4=feedback(X6,1)
figure()
stepplot(System4);
title('PID Step plot of System')
```

```
PID =

50 s^2 + 350 s + 300
```

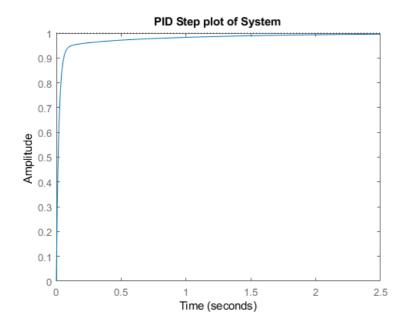
Continuous-time transfer function.

System4 =

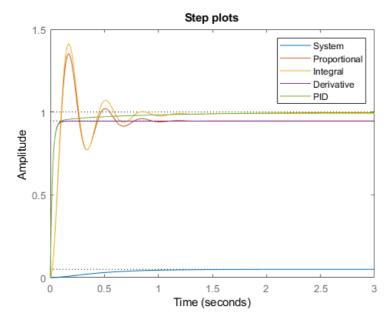
50 s^2 + 350 s + 300

s^3 + 60 s^2 + 370 s + 300

Continuous-time transfer function.



```
figure()
stepplot(G1);
hold on
stepplot(System1);
hold on
stepplot(System2);
hold on
stepplot(System3);
hold on
stepplot(System3);
hold on
stepplot(System3);
hold on
stepplot(System4);
legend("System4,"Proportional","Integral","Derivative","PID")
title("Step plots")
```



Conclusion:

From the experiment, I learnt to plot Step response for P,PI,PD and PID Control Systems with unity feedback.

```
% Further I interpreted :
```

- % Increasing the proportional gain has the effect of proportionally increasing the control signal for the same level of error.
- % The fact that the controller will "push" harder for a given level of error tends to cause the closed-loop system to react more quickly,
- % but also to overshoot more. Another effect of increasing Kp is that it tends to reduce, but not eliminate, the steady-state error.
- % The addition of a derivative term to the controller adds the ability of the controller to "anticipate" error.
- % With simple proportional control, if is fixed, the only way that the control will increase is if the error increases.
- % With derivative control, the control signal can become large if the error begins sloping upward, even while the magnitude of the error
- % is still relatively small. This anticipation tends to add damping to the system, thereby decreasing overshoot.
- % The addition of a derivative term, however, has no effect on the steady-state error.
- % The addition of an integral term to the controller (K_i) tends to help reduce steady-state error.
- % If there is a persistent, steady error, the integrator builds and builds, thereby increasing the
- % control signal and driving the error down. A drawback of the integral term,
- $\ensuremath{\mathrm{\%}}$ however, is that it can make the system more sluggish (and oscillatory)
- % since when the error signal changes sign, it may take a while for the integrator to "unwind."

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