

Solutions Manual

Third
Edition

THEORY OF VIBRATION WITH APPLICATIONS

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PREFACE

This teacher's manual was assembled by the author to aid instructors teaching a course in Vibration from the author's text. There are 517 problems with discussions and solutions presented in this manual.

The author considers problem solving to be a major part of the learning process. It is only through involvement with carefully selected problems that the student will acquire a full understanding of the subject.

A variety of analytical procedures and computational methods are illustrated through these problems. The instructor will find that they emphasize and extend the scope of the text, and discussion of some of these problems in class will greatly aid the student.

The manual represents considerable effort on the part of the author, and although there is reasonable assurance of the correctness of the solutions presented, some errors could be expected. It is hoped that the intended use of the manual strictly by the instructor will be adhered to rigidly.

William T. Thomson
William T. Thomson

1-1

$$x = A \sin \omega t$$

$$A = 0.20 \text{ cm} \quad T = 0.15 \text{ s}$$

$$\omega = \frac{2\pi}{T} = 41.89 \text{ rad/s}$$

$$\dot{x} = \omega A \cos \omega t$$

$$\dot{x}_{\max} = \omega A = 8.38 \text{ cm/s}$$

$$\ddot{x} = -\omega^2 A \sin \omega t$$

$$\ddot{x}_{\max} = \omega^2 A = 350.9 \text{ cm/s}^2$$

1-2

$$\omega = 2\pi f = 2\pi \times 82 = 515.2 \text{ rad/s}$$

$$\omega^2 = 0.2655 \times 10^6$$

$$g = 980.4 \text{ cm/s}^2$$

$$x_{\max} = \ddot{x}_{\max} / \omega^2 = \frac{50 \times 980.4}{0.2655 \times 10^6} = 0.184 \text{ cm}$$

1-3

$$\omega = 2\pi f = 2\pi \times 10 = 62.83 \text{ rad/s}$$

$$T = \frac{1}{f} = 0.10 \text{ s}$$

$$\dot{x}_{\max} = \omega A = 4.57 \text{ m/s}$$

$$A = 0.07274 \text{ m}$$

$$= 7.274 \text{ cm}$$

$$\ddot{x}_{\max} = \omega^2 A = 287.1 \text{ m/s}^2$$

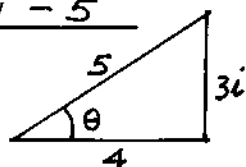
1-4

$$x = A (\sin \omega_1 t + \sin \omega_2 t) = 2A \cos \frac{1}{2}(\omega_1 - \omega_2)t \cdot \sin \frac{1}{2}(\omega_1 + \omega_2)t$$

$$\text{let } \omega_1 = \omega, \quad \omega_2 = \omega + \Delta\omega, \quad \omega_1 + \omega_2 = 2\omega + \Delta\omega \approx 2\omega$$

$$\therefore x \approx 2A \cos \frac{1}{2} \Delta\omega t \cdot \sin \omega t$$

1-5



$$z = 4 + 3i = 5(\cos \theta + i \sin \theta) = 5e^{i\theta}$$

$$\theta = \tan^{-1} \frac{3}{4} = 36^\circ 52' = 0.6435 \text{ rad}$$

1-6

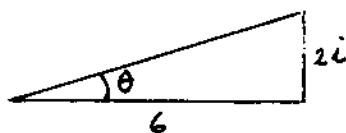
$$\begin{array}{r} 2 + 3i \\ 4 - i \\ \hline \text{Sum} = 6 + 2i \end{array}$$

$$z = A e^{i\theta} \quad A = \sqrt{6^2 + 2^2} = 6.325$$

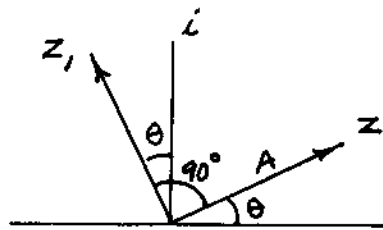
$$\theta = \tan^{-1} \frac{2}{6} = 18^\circ 26' = 0.3217 \text{ rad}$$

$$z = 6.325 e^{0.3217i}$$

$$= 6.325 \angle 18^\circ 26'$$



1-7

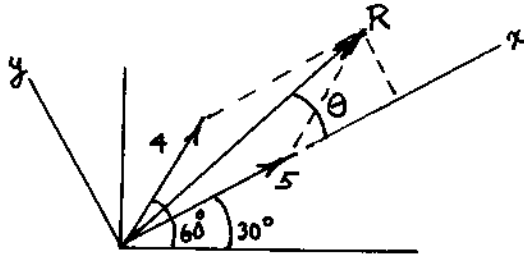


$$z = A (\cos \theta + i \sin \theta) = A e^{i\theta}$$

$$iz = A (i \cos \theta - \sin \theta) = z_1$$

$$= A [\cos(\theta + 90) + i \sin(\theta + 90)]$$

1-8



$$\frac{\pi}{6} = 30^\circ$$

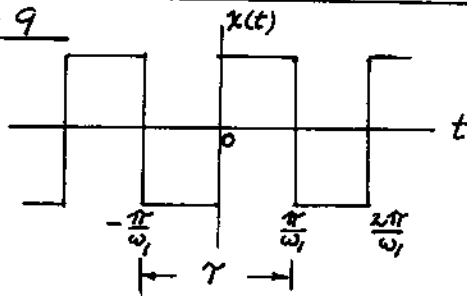
$$R_x = 5 + 4 \cos 30^\circ = 5 + 3.47 = 8.47$$

$$R_y = 4 \sin 30^\circ = 2.00$$

$$R = \sqrt{8.47^2 + 2.0^2} = 8.70$$

$$\theta = \tan^{-1} \frac{2}{8.70} = 12^\circ 57'$$

1-9



$x(t)$ is odd function $\therefore a_n = 0$

$$b_n = \frac{2}{T} \int_{-T/2}^{T/2} x(t) \sin n\omega_1 t dt$$

$$T = \frac{2\pi}{\omega_1}$$

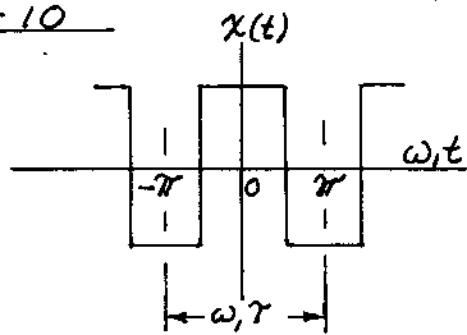
$$b_n = \frac{\omega_1}{\pi} \left[\int_{-\pi/\omega_1}^0 (-1) \sin n\omega_1 t dt + \int_0^{\pi/\omega_1} (+1) \sin n\omega_1 t dt \right]$$

$$= \frac{\omega_1}{\pi} \left[\frac{\cos n\omega_1 t}{n\omega_1} \Big|_{-\pi/\omega_1}^0 - \frac{\cos n\omega_1 t}{n\omega_1} \Big|_0^{\pi/\omega_1} \right] = \frac{2}{n\pi} (1 - \cos n\pi)$$

$$\therefore b_n = \begin{cases} 0 & \text{for } n \text{ even} \\ \frac{4}{n\pi} & \text{" } n \text{ odd} \end{cases}$$

$$x(t) = \frac{4}{\pi} \left(\sin \omega_1 t + \frac{1}{3} \sin 3\omega_1 t + \frac{1}{5} \sin 5\omega_1 t + \dots \right)$$

1-10



$x(t)$ is even function

$$b_m = 0$$

$$a_m = \frac{2}{\gamma} \int_{-\gamma/2}^{\gamma/2} x(t) \cos m\omega_1 t dt$$

$$\gamma = \frac{2\pi}{\omega_1}$$

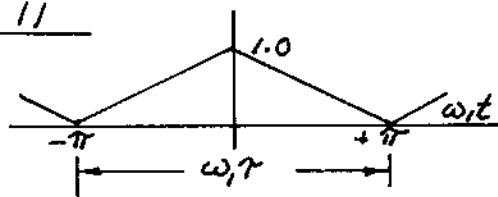
$$\omega_m = m\omega_1$$

$$a_m = \frac{1}{\pi} \int_{-\pi}^{-\pi/2} (-1) \cos n\omega_1 t d(\omega_1 t) + \frac{1}{\pi} \int_{-\pi/2}^{+\pi/2} (+1) \cos n\omega_1 t d(\omega_1 t) + \frac{1}{\pi} \int_{\pi/2}^{\pi} (-1) \cos n\omega_1 t d(\omega_1 t)$$

$$= \frac{1}{n\pi} \left\{ \pm 4 \right\} \quad \begin{array}{l} + \text{ for } n = 1, 5, 9, \dots \\ - \text{ for } n = 3, 7, 11, \dots \end{array}$$

$$x(t) = \frac{4}{\pi} \left(\cos \omega_1 t - \frac{1}{3} \cos 3\omega_1 t + \frac{1}{5} \cos 5\omega_1 t - \dots \right)$$

1-11



$x(t)$ is even function $b_n = 0$

$$x(t) = \begin{cases} \frac{1}{\pi}(t+\pi) & -\pi \leq t \leq 0 \\ \frac{1}{\pi}(\pi-t) & 0 \leq t \leq \pi \end{cases}$$

$$\frac{1}{2} a_0 = \text{average value} = \frac{1}{2}$$

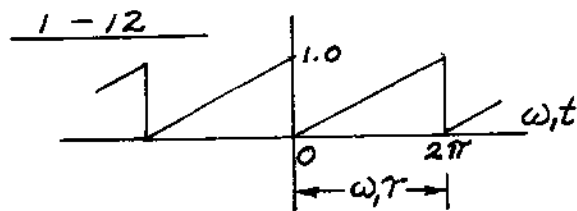
$$a_m = \frac{1}{\pi} \int_{-\pi}^0 \frac{1}{\pi}(t+\pi) \cos n\omega_1 t d(\omega_1 t) + \frac{1}{\pi} \int_0^{\pi} \frac{1}{\pi}(\pi-t) \cos n\omega_1 t d(\omega_1 t)$$

$$= \frac{2}{\pi} \int_0^{\pi} \frac{1}{\pi}(\pi-t) \cos n\omega_1 t d(\omega_1 t) \quad \text{because of symmetry}$$

$$= \frac{2}{\pi} \left. \frac{\sin n\omega_1 t}{n} \right|_0^{\omega_1 t = \pi} - \frac{2}{\pi^2} \left(\frac{\cos n\omega_1 t}{n^2} + \omega_1 t \frac{\sin n\omega_1 t}{n} \right) \Big|_0^{\omega_1 t = \pi}$$

$$= \begin{cases} 0 & \text{for } n \text{ even} \\ \frac{4}{n^2 \pi^2} & \text{for } n \text{ odd} \end{cases}$$

$$x(t) = \frac{1}{2} + \frac{4}{\pi^2} \left(\cos \omega_1 t + \frac{1}{3^2} \cos 3\omega_1 t + \frac{1}{5^2} \cos 5\omega_1 t + \dots \right)$$



$$x = \frac{\omega_1 t}{2\pi} \quad 0 \leq \omega_1 t \leq 2\pi$$

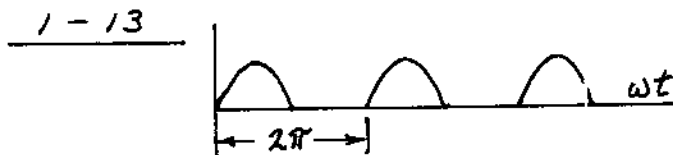
$$C_n = \frac{\omega_1}{2\pi} \int_0^{2\pi} \frac{\omega_1 t}{2\pi} e^{-in\omega_1 t} dt$$

$$C_n = \frac{1}{(2\pi)^2} \int_0^{2\pi} \omega_1 t e^{-in\omega_1 t} d(\omega_1 t) = \frac{1}{(2\pi)^2} \left[\frac{e^{-in\theta}}{(-in)^2} (-in\theta - 1) \right]_0^{2\pi} \quad \text{where } \theta = \omega_1 t$$

$$= \frac{1}{(2\pi)^2 n^2} \left[-1 + (1 + i2\pi n) e^{-i2\pi n} \right] = \frac{i}{2\pi} \frac{1}{n}$$

$$x(t) = C_0 + \frac{i}{2\pi} \left[(e^{i\omega_1 t} - e^{-i\omega_1 t}) + \frac{1}{2} (e^{i2\omega_1 t} - e^{-i2\omega_1 t}) + \frac{1}{3} (e^{i3\omega_1 t} - e^{-i3\omega_1 t}) + \dots \right]$$

$$= \frac{1}{2} - \frac{1}{\pi} \left[\sin \omega_1 t + \frac{1}{2} \sin 2\omega_1 t + \frac{1}{3} \sin 3\omega_1 t + \dots \right]$$



$$\overline{x^2} = \lim_{T \rightarrow \infty} \frac{1}{T} \int_0^{T/2} A^2 \sin^2 \omega t dt = \lim_{T \rightarrow \infty} \frac{A^2}{T} \int_0^{T/2} \frac{1}{2} (1 - \cos 2\omega t) dt$$

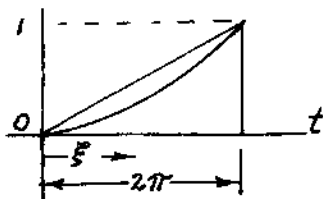
$$= \lim_{T \rightarrow \infty} \frac{A^2}{2} \left(\frac{t}{2} - \frac{\sin 2\omega t}{4\omega} \right) \Big|_0^{T/2} = \lim_{T \rightarrow \infty} \left(\frac{A^2}{4} - \frac{\sin \omega T}{\omega T} \right) = \frac{A^2}{4}$$

$$\therefore x_{RMS} = \sqrt{\overline{x^2}} = \frac{A}{2}$$

1-14

$$x(t) = \frac{\xi}{2\pi}$$

$$x^2(t) = \frac{1}{4\pi^2} \xi^2 \quad 0 \leq \xi \leq 2\pi$$



$$\overline{x^2} = \frac{1}{2\pi} \int_0^{2\pi} \left(\frac{1}{4\pi^2} \xi^2 \right) d\xi = \frac{1}{2\pi} \left(\frac{1}{4\pi^2} \frac{\xi^3}{3} \right) \Big|_0^{2\pi} = \frac{1}{2\pi} \left(\frac{2\pi^3}{3} \right) = \frac{1}{3}$$

1-14 cont.

F.S. of saw tooth wave =

$$x(t) = \frac{1}{2} - \frac{1}{\pi} \left(\sin \omega_1 t + \frac{1}{2} \sin 2\omega_1 t + \frac{1}{3} \sin 3\omega_1 t + \dots \right)$$

$$\begin{aligned} x^2(t) &= \frac{1}{4} - \frac{1}{\pi} \left(\sin \omega_1 t + \frac{1}{2} \sin 2\omega_1 t + \dots \right) \\ &\quad + \frac{1}{\pi^2} \left(\sin \omega_1 t + \frac{1}{2} \sin 2\omega_1 t + \dots \right)^2 \\ &= \frac{1}{4} - \frac{1}{\pi} \left(\sin \omega_1 t + \frac{1}{2} \sin 2\omega_1 t + \dots \right) \end{aligned}$$

$$+ \frac{1}{\pi^2} \left(\sin^2 \omega_1 t + \sin^2 2\omega_1 t + \dots \right) + \text{cross products which will integrate to zero}$$

$$\begin{aligned} \frac{1}{T} \int_0^T x^2(t) dt &= \frac{1}{4} + \frac{1}{\pi \omega_1 T} \cos \omega_1 t \Big|_0^T + \frac{1}{4\pi \omega_1 T} \cos 2\omega_1 t \Big|_0^T + \frac{1}{9\pi \omega_1 T} \cos 3\omega_1 t \Big|_0^T \\ &\quad + \frac{1}{\pi^2 T} \frac{1}{2} \left(t - \frac{\sin 2\omega_1 t}{2\omega_1} \right) \Big|_0^T + \frac{1}{4\pi^2 T} \frac{1}{2} \left(t - \frac{\sin 4\omega_1 t}{4\omega_1} \right) \Big|_0^T + \dots \end{aligned}$$

Let $\omega_1 T = 2\pi k$ where k is an integer $\rightarrow \infty$

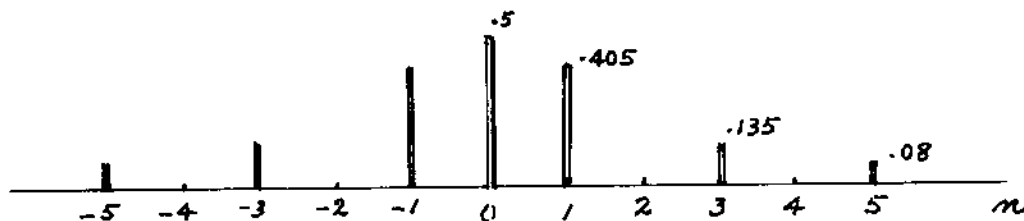
$$\begin{aligned} \overline{x^2} &= \lim_{k \rightarrow \infty} \frac{1}{2\pi k} \int_0^{2\pi k} x^2(t) dt = \frac{1}{4} + \frac{1}{2\pi^2} + \frac{1}{2} \frac{1}{(2\pi)^2} + \frac{1}{2} \frac{1}{(3\pi)^2} + \dots \\ &= \frac{1}{4} + \frac{1}{2\pi^2} \left(1 + \frac{1}{4} + \frac{1}{9} + \frac{1}{16} + \dots \right) = \frac{1}{3} \end{aligned}$$

1-15

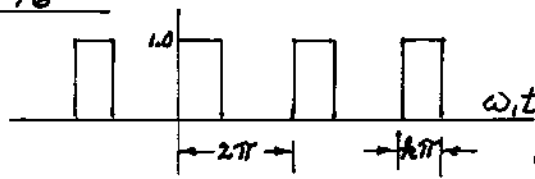
$$x(t) = \frac{1}{2} + \frac{4}{\pi^2} \left(\cos \omega_1 t + \frac{1}{3} \cos 3\omega_1 t + \frac{1}{5} \cos 5\omega_1 t + \dots \right)$$

Fourier Spectrum = plot of coefficients. For this case $b_n = 0$

$$C_n = \sqrt{a_n^2 + b_n^2} = a_n \quad C_0 = \frac{a_0}{2}$$



1-16



$$k = \frac{2}{3}$$

$$C_0 = \frac{A_0}{2} = \text{average value} = \frac{k\pi}{2\pi} = \frac{k}{2} = \frac{1}{3}$$

$$a_m = \frac{1}{m\pi} \sin m k \pi = \frac{1}{m\pi} \sin \frac{n}{3} 2\pi$$

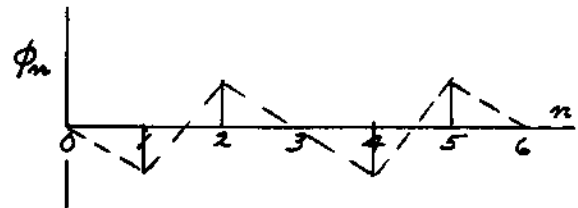
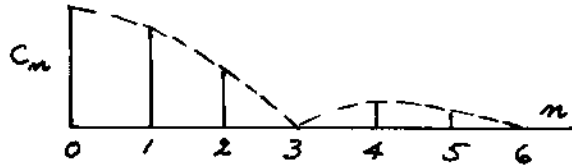
$$b_m = \frac{1}{m\pi} (1 - \cos m k \pi) = \frac{1}{m\pi} (1 - \cos \frac{n}{3} 2\pi)$$

$$2C_m = \sqrt{a_m^2 + b_m^2} = \frac{\sqrt{2}}{n\pi} \sqrt{(1 - \cos \frac{n}{3} 2\pi)}$$

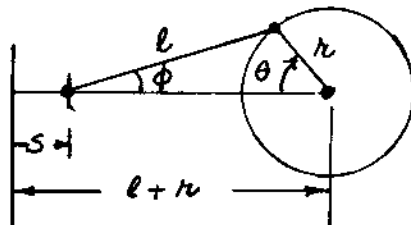
$$\phi_m = \tan^{-1} \frac{1 - \cos \frac{n}{3} 2\pi}{\sin \frac{n}{3} 2\pi}$$

n	C_m	ϕ_m
1	.2758	-60°
2	.1379	60°
3	0	0

n	C_m	ϕ_m
4	.0689	-60°
5	.0552	60°
6	0	0



1-17



$$l + r - S = r \cos \theta + l \cos \phi$$

$$l \sin \phi = r \sin \theta$$

$$\therefore \cos \phi = \left[1 - \left(\frac{r}{l} \right)^2 \sin^2 \theta \right]^{1/2}$$

$$= 1 - \frac{1}{2} \left(\frac{r}{l} \right)^2 \sin^2 \theta - \frac{1}{8} \left(\frac{r}{l} \right)^4 \sin^4 \theta - \dots$$

$$S = r \left[1 - \cos \theta + \frac{1}{2} \left(\frac{r}{l} \right) \sin^2 \theta + \frac{1}{8} \left(\frac{r}{l} \right)^3 \sin^4 \theta + \dots \right]$$

$$\text{using } \sin^2 \theta = \frac{1}{2} (1 - \cos 2\theta), \quad \sin^4 \theta = \frac{1}{4} \left(\frac{3}{2} - 2 \cos 2\theta + \frac{1}{2} \cos 4\theta \right)$$

$$S = r \left[1 - \cos \theta + \frac{1}{2} \left(\frac{r}{l} \right) \frac{1}{2} (1 - \cos 2\theta) + \dots \right]$$

$$= r \left[1 + \frac{1}{4} \left(\frac{r}{l} \right) - \cos \theta - \frac{1}{4} \left(\frac{r}{l} \right) \cos 2\theta + \dots \right] \quad \text{which retains only } \left(\frac{r}{l} \right) \text{ to first power}$$

$$\text{Ratio of 2nd harmonic / 1st harmonic} = \frac{1}{4} \left(\frac{r}{l} \right) = \frac{1}{12}$$

$$\frac{1-18}{\overline{x^2}} = \lim_{T \rightarrow \infty} \frac{1}{T} \int_0^T x^2(t) dt = \frac{A^2 \frac{1}{2} \pi}{\pi} = \frac{1}{2} A^2 = 0.10 A^2$$

$$r.m.s = \sqrt{\overline{x^2}} = 0.3162 A$$

$$\frac{1-19}{x = \left(1 - \frac{t}{\pi}\right) \quad 0 \leq t \leq \pi}$$

$$x^2 = 1 - \frac{2t}{\pi} + \frac{t^2}{\pi^2}$$

$$\overline{x^2} = \frac{1}{\pi} \int_0^{\pi} \left(1 - \frac{2t}{\pi} + \frac{t^2}{\pi^2}\right) dt = \frac{1}{3}$$

$$\frac{1-20}{}$$

$$Db = 20 \log_{10} \left(\frac{x_1}{x_2} \right) = 0.50$$

$$\log_{10} \left(\frac{x_1}{x_2} \right) = \frac{0.50}{20} = 0.0250$$

$$\left(\frac{x_1}{x_2} \right) = 10^{0.0250} = 1.0593$$

$$x_1 = 1.0593 x_2 = 1.0593 \times 2.5 \text{ mm} = 2.6481$$

$$\text{Error} = 0.0593 \times 2.5 \text{ mm} = \pm 0.148 \text{ mm}$$

$$\frac{1-21}{}$$

$$Db = 20 \log_{10} (10) = 20$$

$$Db = 20 \log_{10} (50) = 33.98$$

$$Db = 20 \log_{10} (100) = 40.0$$

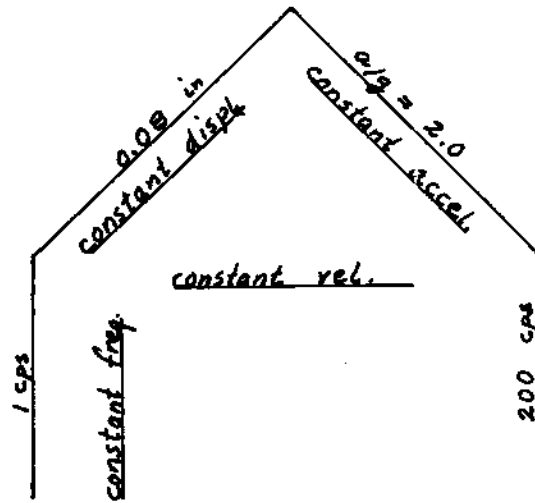
$$\frac{1-22}{}$$

$$Db = 20 \log_{10} \left(\frac{x_p}{x_{1000}} \right) = 32.$$

$$\log_{10} \left(\frac{x_p}{x_{1000}} \right) = \frac{32}{20} = 1.60$$

$$\frac{x_p}{x_{1000}} = 10^{1.60} = 39.8$$

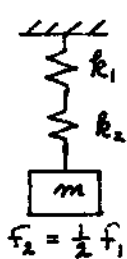
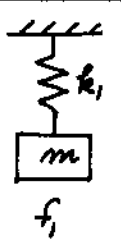
1-23



2-1 From Eq. 2.2-9

$$f = \frac{15.76}{\sqrt{\Delta_{mm}}} = \frac{15.76}{\sqrt{7.87}} = 5.62 \text{ Hz}$$

2-2



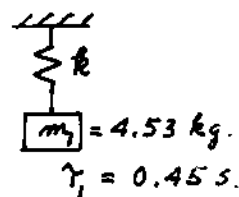
$$f_1 = \sqrt{\frac{k_1}{m}}$$

$$f_2 = \sqrt{\frac{k_1, k_2}{m(k_1 + k_2)}}$$

$$\frac{1}{2} \sqrt{\frac{k_1}{m}} = \sqrt{\frac{k_1, k_2}{m(k_1 + k_2)}}$$

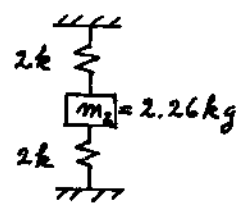
$$\frac{1}{4} k_1 = \frac{k_1 k_2}{k_1 + k_2} \quad \therefore k_2 = \frac{1}{3} k_1$$

2-3



$$k = \left(\frac{2\pi}{T_1} \right)^2 m_1 = \left(\frac{2\pi}{0.45} \right)^2 4.53$$

$$= 883.5 \text{ N/m}$$



$$T_2 = 2\pi \sqrt{\frac{m_2}{4k}} = 2\pi \sqrt{\frac{2.26}{4 \times 883.5}}$$

$$= 0.159 \text{ s}$$

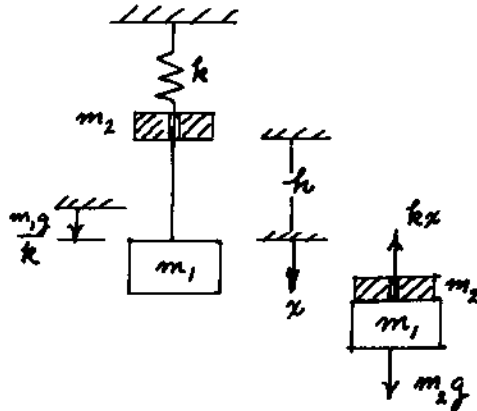
2-4

$$\frac{k}{m} = (2\pi f)^2 = \left(2\pi \frac{94}{60}\right)^2 \quad \frac{k}{m+0.453} = \left(2\pi \frac{76.7}{60}\right)^2$$

$$\frac{m+0.453}{m} = \left(\frac{94}{76.7}\right)^2 \quad \therefore m = 0.9028 \text{ kg}$$

$$k = 87.48 \text{ N/m}$$

2-5



x measured from static equilibrium position of m_1, k

Eq. of motion after impact

$$(m_1+m_2)\ddot{x} = -kx + m_2g$$

Gen solution:

$$x(t) = \frac{m_2g}{k} + A\sin\omega t + B\cos\omega t$$

Initial conditions

$$x(0) = 0 = \frac{m_2g}{k} + B \quad \therefore B = -\frac{m_2g}{k}$$

$$\dot{x}(0) = \frac{m_2\sqrt{2gh}}{m_1+m_2} = \omega A \quad \therefore A = \frac{m_2\sqrt{2gh}}{(m_1+m_2)\omega}$$

$$\omega = \sqrt{\frac{k}{m_1+m_2}}$$

$$\begin{aligned} \therefore x(t) &= \frac{m_2g}{k} + \frac{m_2\sqrt{2gh}}{m_1+m_2} \sqrt{\frac{m_1+m_2}{k}} \sin\omega t - \frac{m_2g}{k} \cos\omega t \\ &= \frac{m_2g}{k}(1 - \cos\omega t) + \frac{m_2\sqrt{2gh}}{\sqrt{k(m_1+m_2)}} \sin\omega t \end{aligned}$$

2-6

$$\omega_n^2 = \frac{k}{m} = 4.0 \quad \omega_n = 2.0$$

$$x = x_0 \cos\omega t + \frac{v_0}{\omega} \sin\omega t = 2\cos 2t - \frac{8}{2} \sin 2t$$

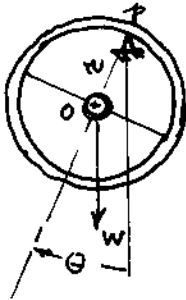
$$\dot{x} = -4\sin 2t - 8\cos 2t = 0 \quad \therefore \tan 2t_p = -2$$

$$\therefore 2t_p = 116.57^\circ \quad \sin 116.57^\circ = 0.8944 \quad \cos 116.57^\circ = -0.4472$$

$$x_{\max} = 2(-0.4472) - 4(0.8944) = -4.472 \text{ cm}$$

$$\ddot{x}_{\max} = \omega^2 x_{\max} = 4(\pm 4.472) = \pm 17.89 \text{ cm/s}^2$$

2-7



$$J_P \ddot{\theta} = -Wr \sin \theta$$

$$\ddot{\theta} = -\omega^2 \theta$$

$$J_P = \frac{Wr}{\omega^2} = \frac{70 \times 6}{\left(\frac{2\pi}{1.22}\right)^2} = 15.83$$

$$J_O = J_P - \frac{W}{g} r^2 = 15.83 - \frac{70}{386} \times 6^2 = 9.30 \text{ lb in sec}^2$$

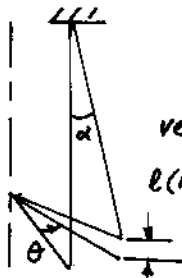
2-8

$$\omega = 2\pi \frac{53}{60} = 5.55 \text{ rad/s}$$

$$J_P = \frac{Wr}{\omega^2} = \frac{21.35 \times 0.254}{5.55^2} = 0.1761$$

$$J_{CG} = J_O - \frac{W}{g} r^2 = 0.1761 - \frac{21.35 \times 0.254^2}{9.81} = 0.0356 \text{ kg m}^2$$

2-9



$$r\theta = l\alpha$$

$$\alpha = \frac{r\theta}{l}$$

vertical displ =

$$l(1 - \cos \theta) = l\left[1 - \left(1 - \frac{1}{2}\alpha^2 + \dots\right)\right] = \frac{l\alpha^2}{2} = \frac{l}{2} \left(\frac{r\theta}{l}\right)^2$$

Work done = change in KE

$$W \frac{l}{2} \frac{r^2 \theta_{\max}^2}{l^2} = \frac{1}{2} J \dot{\theta}_{\max}^2 = \frac{1}{2} J \omega^2 \theta_{\max}^2$$

$$J = \frac{W}{g} k^2 \quad k = \text{rad. of gyr.}$$

$$W \frac{l}{2} = \frac{W}{g} k^2 \omega^2$$

$$k = \frac{r}{\omega} \sqrt{\frac{g}{l}} = \frac{0.254 \times 2.17}{2\pi} \sqrt{\frac{9.81}{1.829}} = 0.2032$$

$$k = 4507 \text{ mm}$$

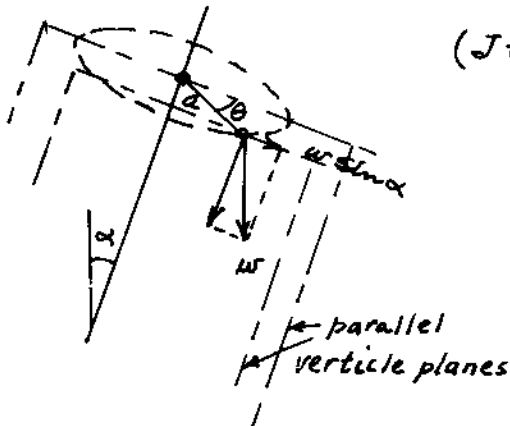
2-10

Moment about shaft = $(a \sin \theta) w \sin \alpha$

$$\left(J + \frac{w}{g} a^2\right) \ddot{\theta} = -(a \sin \theta) w \sin \alpha$$

$$\approx -(a w \sin \alpha) \theta$$

$$\therefore f_n = \frac{1}{2\pi} \sqrt{\frac{w a \sin \alpha}{J + \frac{w}{g} a^2}}$$



2-11

$$T = \frac{1}{2} m \dot{x}^2 + \frac{1}{2} J_o \dot{\theta}^2 \quad k\dot{\theta} = \dot{x}$$

$$= \frac{1}{2} \left(m + \frac{J_o}{R^2} \right) \dot{x}^2 \quad \dot{x} = \omega x$$

$$U = \frac{1}{2} k x^2 \quad \therefore \omega = \sqrt{\frac{k}{m + J_o/R^2}}$$

2-12

$$\gamma = 2\pi \sqrt{\frac{L}{g}}$$

$$L = g \left(\frac{\gamma}{2\pi} \right)^2 = 9.81 \left(\frac{2}{2\pi} \right)^2 = 0.994 \text{ m}$$

$$v_{\max} = L(\omega \theta_0) = \frac{.003175}{.01} \text{ m/s}$$

$$\theta_0 = \frac{.3175}{.994 \pi} = .1017 \text{ rad} = 5.826^\circ$$

2-13

$$\text{water weighs } 9802 \text{ N/m}^3 \quad \therefore \rho = 1.2 \times 9802 = 11762 \text{ N/m}^3$$

$$\text{buoyant force} = \pi R^2 x \cdot \rho = m \ddot{x} = \omega^2 x$$

$$\frac{1}{\omega} = \frac{\gamma}{2\pi} = \sqrt{\frac{m}{\pi R^2 \rho}}$$

$$m = .0372 \text{ kg}$$

$$R = .0032 \text{ m}$$

$$\gamma = 2\pi \sqrt{\frac{.0372}{\pi \times .0032^2 \times 11762}} = 1.97 \text{ s}$$

2-14

moment about geom. center

$$-W \delta \theta = J_o \ddot{\theta} = -\omega^2 J_o \theta$$

$$J_o = \frac{8W}{\omega^2} = \frac{8W (1.3)^2}{(2\pi)^2} = 0.3428 W$$

2-15

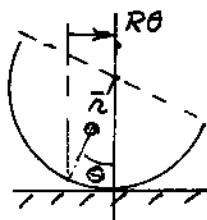
$$-W h \theta = J \ddot{\theta} = -\omega^2 J \theta$$

$$\frac{1}{\omega} = \frac{\gamma}{2\pi} = \sqrt{\frac{J}{Wh}}$$

$$\gamma = 2\pi \sqrt{\frac{J}{Wh}}$$

2-16Displ. of cg = $(R - \bar{r})\theta$

$$T_{\max} = U_{\max}$$



$$T_{\max} = \frac{1}{2} m (R - \bar{r})^2 \dot{\theta}_{\max}^2 + \frac{1}{2} J_{cg} \dot{\theta}_{\max}^2$$

$$= \frac{1}{2} m [(R - \bar{r})^2 + (R^2 - \bar{r}^2)] \omega^2 \theta_{\max}^2$$

$$U_{\max} = mg \bar{r} (1 - \cos \theta_{\max}) \approx mg \bar{r} \frac{\theta_{\max}^2}{2}$$

2-16 cont

$$T_{max} = U_{max}$$

$$\omega^2 = \frac{\bar{h}g}{(R-\bar{h})^2 + (R^2 - \bar{h}^2)} = \frac{\bar{h}g}{2R(R-\bar{h})}$$

$$\gamma = 2\pi \sqrt{\frac{2R(R-\bar{h})}{\bar{h}g}} \quad \text{but } \bar{h} = \frac{2R}{\pi} \quad \therefore \gamma = 2\pi \sqrt{\frac{R(\pi-2)}{g}}$$

2-17

$$U = mgh(1 - \cos \phi) \approx mgh \frac{1}{2} \phi^2 \quad h\phi = \frac{a}{2} \theta$$

$$= mgh \frac{h}{2} \left(\frac{a\theta}{2h} \right)^2 = mgh \frac{a^2}{8} \frac{\theta^2}{h}$$

$$T = \frac{1}{2} \left(m \frac{L^2}{12} \right) \dot{\theta}^2 = \frac{1}{2} \left(m \frac{L^2}{12} \right) \omega^2 \theta^2$$

$$T_{max} = U_{max} \quad \therefore \gamma = 2\pi \frac{L}{a} \sqrt{\frac{h}{3g}}$$

2-18

$$\gamma_1 = 2\pi \sqrt{\frac{h}{g}}$$

$$\text{for } \gamma_2 \quad T = \frac{1}{2} m k^2 \omega^2 \theta^2$$

$$U = mgh \frac{L^2}{8} \frac{\theta^2}{h}$$

$$\therefore \gamma_2 = 2\pi \sqrt{\frac{4hk^2}{gL^2}}$$

$$k = \frac{\gamma_2 L}{2\pi} \sqrt{\frac{g}{4h}} = \frac{\gamma_2}{\gamma_1} \left(\frac{L}{2} \right)$$

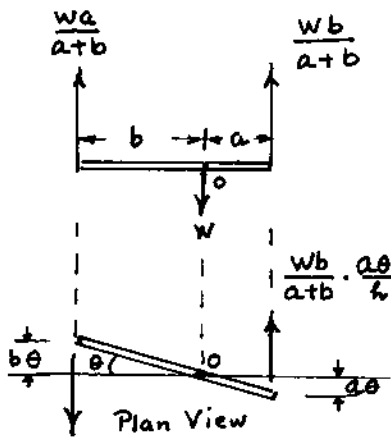
2-19

$$\sum M_o = J \ddot{\theta} = - \frac{Wba}{a+b} \frac{\theta \cdot a}{h} - \frac{Wab}{a+b} \frac{\theta \cdot b}{h}$$

$$\frac{W}{g} k^2 \ddot{\theta} + \frac{Wab}{h} \theta = 0$$

$$\ddot{\theta} + \left(\frac{gab}{k^2 h} \right) \theta = 0$$

$$\therefore f = \frac{1}{2\pi} \sqrt{\frac{gab}{k^2 h}}$$



$$\frac{Wa}{a+b} \cdot \frac{b\theta}{h}$$

$$\therefore \sum F_{horiz} = 0$$

2-20

$$J_o \text{ of wheel about torsion bar} = J_{cm} + m(24'')^2$$

$$= m(k^2 + 24^2) = m(9^2 + 24^2) = 657 m$$

$$\text{Stiffness of torsion bar } K = \frac{GI_p}{l}$$

$$I_p = \frac{\pi D^4}{32} = \frac{\pi (1.50)^4}{32} = 0.497 \text{ in}^4 = \text{polar mom. inertia of torsion bar}$$

$$G = 11.2 \times 10^6 \text{ lb/in}^2 = \text{shear modulus of steel}$$

$$K = \frac{(11.2 \times 10^6) \times (0.497)}{50} = 0.1113 \times 10^6 \text{ lb.in./rad.}$$

$$J_o \ddot{\theta} + K \theta = 0 \quad f = \frac{1}{2\pi} \sqrt{\frac{K}{J_o}} = \frac{1}{2\pi} \sqrt{\frac{0.1113 \times 10^6 \times 386}{38 \times 657}} = 6.60 \text{ cps}$$

locked wheel.

$$\text{With wheel free } J_o = m(24)^2 = 576 m$$

$$f = \frac{1}{2\pi} \sqrt{\frac{0.1113 \times 10^6 \times 386}{38 \times 576}} = 7.05 \text{ c.p.s.}$$

2-21

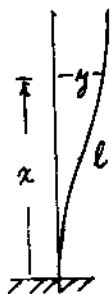
$$\frac{l\rho}{g} \ddot{x} = -2x\rho \quad \ddot{x} + \frac{2g}{l} x = 0$$

$$\omega^2 = \frac{2g}{l} \quad \gamma = 2\pi \sqrt{\frac{l}{2g}}$$

2-22

$$k = 2 \left(\frac{12EI}{l^3} \right) \quad \gamma = 2\pi \sqrt{\frac{m}{k}} = 2\pi \sqrt{\frac{m l^3}{24EI}}$$

2-23



$$y = \frac{1}{2} y_{\max} (1 - \cos \frac{\pi x}{l}) \sin \omega t$$

$$\dot{y} = \frac{1}{2} \omega y_{\max} (1 - \cos \frac{\pi x}{l}) \cos \omega t$$

$$T = \frac{1}{2} \int_0^l m(x) \dot{y}^2 dx = \frac{1}{2} \frac{m}{4} y_{\max}^2 \int_0^l (1 - \cos \frac{\pi x}{l})^2 \omega^2 dx \cos^2 \omega t$$

$$= \frac{1}{2} \cdot \frac{m}{4} y_{\max}^2 \omega^2 \cos^2 \omega t \int_0^l (1 - 2 \cos \frac{\pi x}{l} + \cos^2 \frac{\pi x}{l}) dx$$

$$= \quad \quad \quad \left[x - \frac{2l}{\pi} \sin \frac{\pi x}{l} + \frac{x}{2} + \frac{1}{2} \cdot \frac{l}{2\pi} \sin \frac{2\pi x}{l} \right]_0^l$$

2-23 Cont.

$$T = \quad \quad \left[\frac{3}{2} \ell - 0 + 0 \right]$$

$$= \frac{1}{2} \left(\frac{m}{4} \cdot \frac{3\ell}{2} \right) \omega^2 y_{\max}^2 \cos^2 \omega t$$

$\therefore m_{\text{eff}} = \left(\frac{3}{8} m\ell \right)$ for each column, where
 $m\ell = \text{total mass of each column}$

2-24

$$T = \frac{1}{2} m \dot{x}^2 + \frac{1}{2} J \left(\frac{\dot{x}}{b} \right)^2 \quad \text{where } J = \text{moment of inertia of linkage about pivot.}$$

$$x = \frac{b}{a} x_m$$

$$T = \frac{1}{2} m \left(\frac{b}{a} \right)^2 \dot{x}_m^2 + \frac{1}{2} J \left(\frac{b}{a} \right)^2 \frac{1}{b^2} \dot{x}_m^2$$

$$= \frac{1}{2} \left[m \left(\frac{b}{a} \right)^2 + \frac{J}{a^2} \right] \dot{x}_m^2 \quad \therefore m_{\text{eff}} = \left[m \left(\frac{b}{a} \right)^2 + \frac{J}{a^2} \right]$$

2-25

$$y = \frac{1}{2} y_0 \left[3 \left(\frac{x}{\ell} \right)^2 - \left(\frac{x}{\ell} \right)^3 \right]$$

$$T = \frac{1}{2} m \int_0^{\ell} \dot{y}^2 dx = \frac{1}{2} \frac{m}{4} \dot{y}_0^2 \int_0^{\ell} \left[9 \left(\frac{x}{\ell} \right)^4 - 6 \left(\frac{x}{\ell} \right)^5 + \left(\frac{x}{\ell} \right)^6 \right] dx$$

$$= \frac{1}{2} m \dot{y}_0^2 \frac{\ell}{4} \left[\frac{9}{5} - 1 + \frac{1}{7} \right] = \frac{1}{2} \left(\frac{33}{140} m\ell \right) \dot{y}_0^2$$

2-26

$$T = \frac{1}{2} [J_0 \dot{\theta}^2 + m_1 (b \dot{\theta})^2] = \frac{1}{2} [J_0 + m_1 b^2] \dot{\theta}^2$$

$$\dot{\theta} = \dot{x}/b \quad T = \frac{1}{2} [J_0/b^2 + m_1] \dot{x}^2 \quad m_{\text{eff}} = J_0/b^2 + m_1$$

2-27

Let $\theta_0 = \text{rotation of } J$, $T_0 = \text{total torque}$
 $T_L = \text{torque to left of } J$
 $T_R = \quad \quad \text{" right " "}$

$$\left(\frac{1}{k_1} + \frac{1}{k_2} \right) T_L = \theta_0 \quad \left(\frac{1}{k_2} \right) T_R = \theta_0$$

2-27 Cont.

$$T_o = T_L + T_R = \left[\frac{1}{\left(\frac{1}{K_1} + \frac{1}{K_2}\right)} + \frac{1}{\left(\frac{1}{K_2}\right)} \right] \theta_o = K \theta_o$$

$$\therefore K = \left(\frac{K_1 K_2}{K_1 + K_2} + K_2 \right) \quad \omega_n = \sqrt{\frac{K}{J}} = \frac{2\pi}{T}$$

2-28

$$T = \frac{1}{2} m_1 \dot{x}^2 + \frac{1}{2} J_2 \left(\frac{\dot{x}}{R_2} \right)^2 + \frac{1}{2} J_3 \left(\frac{R_2}{R_1} \frac{\dot{x}}{R_2} \right)^2$$

$$= \frac{1}{2} \left[m_1 + J_2/R_2^2 + J_3/R_1^2 \right] \dot{x}^2 = \frac{1}{2} m_{\text{eff}} \dot{x}^2$$

$$U = \frac{1}{2} k_1 x^2 + \frac{1}{2} K_3 \left(\frac{x}{R_1} \right)^2 = \frac{1}{2} \left[k_1 + K_3/R_1^2 \right] x^2 = \frac{1}{2} k_{\text{eff}} x^2$$

2-29

$$T = \frac{1}{2} J_1 \dot{\theta}_1^2 + \frac{1}{2} J_2 \left(\frac{R_1}{R_2} \dot{\theta}_1 \right)^2$$

$$= \frac{1}{2} \left[J_1 + J_2 \left(\frac{R_1}{R_2} \right)^2 \right] \dot{\theta}_1^2 = \frac{1}{2} J_{\text{eff}} \dot{\theta}_1^2$$

2-30

$$T = \frac{1}{2} J_1 \dot{\theta}_1^2 + \frac{1}{2} (m_o + m_2) \dot{x}^2 + \frac{1}{2} J_2 \dot{\theta}_2^2$$

$$\dot{\theta}_1 = \dot{x}/R_1 \quad \dot{\theta}_2 = \dot{x}/R_2$$

$$T = \frac{1}{2} \left[J_1/R_1^2 + (m_o + m_2) + J_2/R_2^2 \right] \dot{x}^2$$


$$U = \frac{1}{2} k_1 (R \theta_1)^2 + \frac{1}{2} k_2 (R_2 \theta_2)^2$$

$$= \frac{1}{2} k_1 \left(\frac{R}{R_1} x \right)^2 + \frac{1}{2} k_2 \left(R_2 \frac{x}{R_2} \right)^2$$

$$= \frac{1}{2} \left[k_1 \left(\frac{R}{R_1} \right)^2 + k_2 \right] x^2$$

$$f = \frac{1}{2\pi} \sqrt{\frac{k_1 \left(\frac{R}{R_1} \right)^2 + k_2}{J_1/R_1^2 + (m_o + m_1) + J_2/R_2^2}}$$

2-31 $m_{eff} = M + \frac{33}{140} ml$ (see Prob. 2-25)

 beam vol = $(.1016 \times .635 \times 8.89) = .5735 \text{ cm}^3$
wt. of steel = 0.07655 N/cm^3

wt. of beam = $.5735 \times .07655 = .04390 \text{ N}$

mass of beam = $\frac{.04390}{9.81} = .00475 \text{ kg} = ml$

$\frac{33}{140} ml = .001055$ beam stiffness = $\frac{3EI}{l^3} = k$

$E = 200 \times 10^9 \text{ N/m}^2$ $I = \frac{bh^3}{12} = \frac{.635 \times .1016^3}{12} = .0000553 \times 10^{-8} \text{ m}^4$

$k = \frac{3 \times 200 \times 10^9 \times 553 \times 10^{-15}}{(.0889)^3} = 473.96 \text{ N/m}$

$f = \frac{1}{2\pi} \sqrt{\frac{k}{m_{eff}}}$ $m_{eff} = M + .001055 = \frac{3EI}{l^3 4\pi^2 f^2} = \frac{473.96}{4\pi^2 \times 400}$

$\therefore M = 0.0289 \text{ kg}$

2-32

$C_c = 2 \sqrt{mk} = 2 \sqrt{.907 \times 7 \times 10^2} = 50.4 \frac{\text{Ns}}{\text{m}}$

2-33

$F_d = C v$ $C = \frac{F_d}{v} = \frac{0.50}{1.20} = 0.417 \frac{\text{lb sec}}{\text{in}}$

$C = 0.417 \frac{\text{lb.s}}{\text{in}} \times 4.448 \frac{\text{N}}{\text{lb}} \times \frac{1}{2.54} \frac{\text{in}}{\text{cm}} = 0.7303 \frac{\text{Ns}}{\text{cm}} = 73.03 \frac{\text{Ns}}{\text{m}}$

$\zeta = \frac{C}{C_c} = \frac{73.03}{50.4} = 1.45$

2-34

(a) $\zeta = 2$ Eq.(2.3-18) $A = -B = \frac{v_0}{2\omega_m \sqrt{\zeta^2 - 1}}$

$\frac{x\omega_m}{v_0} = \frac{1}{3.464} (e^{-0.268\omega_m t} - e^{-3.732\omega_m t})$

(b) $\zeta = 0.50$ $\frac{x\omega_m}{v_0} = \frac{e^{-0.50\omega_m t}}{0.865} \sin 0.865\omega_m t$ Eq.(2.3-16)

(c) $\zeta = 1.0$ Eq.(2.3-19) $\frac{x\omega_m}{v_0} = \omega_m t e^{-\omega_m t}$

2-35

$$\delta = \ln \frac{x_1}{x_2} = \ln \frac{1.00}{.980} = \ln 1.020408 = 0.0202$$

$$\zeta \approx \frac{\delta}{2\pi} = .003215$$

$$\omega_n = \sqrt{\frac{k}{m}} = \sqrt{\frac{1750}{2.267}} = 27.78 \approx \omega_d$$

$$c = 2m\omega_n\zeta = 2 \times 2.267 \times 27.78 \times .003215 = .405 \frac{Ns}{m}$$

2-36

$$(a) \quad \zeta = \frac{c}{2m} \sqrt{\frac{m}{k}} = \frac{12.43}{2 \times 4.534} \sqrt{\frac{4.534}{3500}} = 0.0493$$

$$(b) \quad \delta = \frac{2\pi\zeta}{\sqrt{1-\zeta^2}} = 0.3101$$

$$(c) \quad \frac{x_n}{x_{n+1}} = e^{\delta} = (2.718)^{.3101} = 1.364$$

2-37

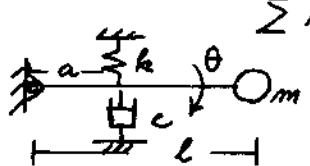
$$(a) \quad \zeta = \frac{c}{2\sqrt{mk}} = \frac{70}{2\sqrt{17.5 \times 7000}} = 0.10$$

$$(b) \quad f_d = \frac{1}{2\pi} \sqrt{1-\zeta^2} \sqrt{\frac{k}{m}} = \frac{1}{2\pi} \sqrt{1-.01} \sqrt{\frac{7000}{17.5}} = 3.167 \text{ Hz}$$

$$(c) \quad \delta = \frac{2\pi\zeta}{\sqrt{1-\zeta^2}} = .6315$$

$$(d) \quad x_n/x_{n+1} = e^{.6315} = 1.874$$

2-38



$$\sum M_o = -ac(a\dot{\theta}) - a k(a\theta) = ml^2 \ddot{\theta}$$

$$\ddot{\theta} + \frac{c}{m} \left(\frac{a}{l}\right)^2 \dot{\theta} + \frac{k}{m} \left(\frac{a}{l}\right)^2 \theta = 0 \quad \text{let } \theta = e^{st}$$

$$s_{1,2} = -\frac{c}{2m} \left(\frac{a}{l}\right)^2 \pm \sqrt{\left(\frac{ca^2}{2ml^2}\right)^2 - \frac{k}{m} \left(\frac{a}{l}\right)^2}$$

$$\text{crit. damp. } \frac{ca^2}{2ml^2} = \frac{a}{l} \sqrt{\frac{k}{m}} \quad c_c = 2 \frac{l}{a} \sqrt{k m}$$

$$\omega_d = \frac{a}{l} \sqrt{\frac{k}{m} - \left(\frac{ca}{2ml}\right)^2} = \frac{a}{l} \sqrt{\frac{k}{m}} \sqrt{1 - \left(\frac{ca}{2l\sqrt{k m}}\right)^2} = \omega_n \sqrt{1-\zeta^2}$$

$$\therefore \omega_n = \frac{a}{l} \sqrt{\frac{k}{m}}, \quad \zeta = \frac{ca}{2l\sqrt{k m}} \quad \text{identify from } \ddot{\theta} + 2\zeta\omega_n \dot{\theta} + \omega_n^2 \theta = 0$$

2-39

$$\Sigma M_o = ma^2 \ddot{\theta} = -kb^2 \theta - ca^2 \dot{\theta}$$

$$\ddot{\theta} + \frac{c}{m} \dot{\theta} + \frac{k}{m} \left(\frac{b}{a}\right)^2 \theta = 0$$

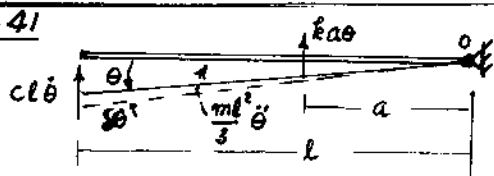
$$\therefore \omega_n = \frac{b}{a} \sqrt{\frac{k}{m}} \quad \omega_d = \sqrt{\frac{k}{m} \left(\frac{b}{a}\right)^2 - \left(\frac{c}{2m}\right)^2} \quad c_c = \frac{2b}{a} \sqrt{km}$$

2-40

$$\delta = \ln \frac{1.0}{0.95} = \ln 1.0527 = .05129$$

$$\delta = \frac{2\pi \zeta}{\sqrt{1-\zeta^2}} = .05129 \quad \zeta = .00816$$

2-41



$$\delta W = -\frac{ml^2}{3} \ddot{\theta} \delta \theta - cl \delta \theta - k a \delta \theta = 0$$

$$\ddot{\theta} + \frac{3c}{m} \dot{\theta} + \frac{3k}{m} \left(\frac{a}{l}\right)^2 \theta = 0$$

$$\ddot{\theta} + 2\zeta \omega_n \dot{\theta} + \omega_n^2 \theta = 0$$

$$\therefore \omega_n = \frac{a}{l} \sqrt{\frac{3k}{m}}, \quad c_c = \frac{2a}{3l} \sqrt{3km}, \quad \zeta = \frac{3}{2} \frac{c}{m \omega_n}$$

$$\omega_d = \omega_n \sqrt{1-\zeta^2} = \frac{a}{l} \sqrt{\frac{3k}{m}} \sqrt{1 - \frac{9}{4} \left(\frac{c}{m \omega_n}\right)^2} = \frac{a}{l} \sqrt{\frac{3k}{m}} \sqrt{1 - \frac{3}{4km} \left(\frac{cl}{a}\right)^2}$$

2-42

$$\frac{W}{g} \ddot{x} + 2\mu A \dot{x} + kx = 0$$

$$\ddot{x} + \frac{2\mu A g}{W} \dot{x} + \frac{k g}{W} x = 0 \quad \therefore \gamma_1 = 2\pi \sqrt{\frac{W}{k g}}$$

$$f_2 = \frac{1}{\gamma_2} = \frac{1}{2\pi} \sqrt{\frac{k g}{W} - \left(\frac{\mu A g}{W}\right)^2} = \frac{1}{2\pi} \sqrt{\left(\frac{2\pi}{\gamma_1}\right)^2 - \left(\frac{\mu A g}{W}\right)^2}$$

square both sides

$$\left(\frac{2\pi}{\gamma_2}\right)^2 - \left(\frac{2\pi}{\gamma_1}\right)^2 = -\left(\frac{\mu A g}{W}\right)^2$$

$$\therefore \mu = \frac{2\pi W}{A g} \sqrt{\frac{\gamma_2^2 - \gamma_1^2}{\gamma_1^2 \gamma_2^2}} = \frac{2\pi W}{A g \gamma_1 \gamma_2} \sqrt{\gamma_2^2 - \gamma_1^2}$$

2-43

$$\omega_n = \sqrt{\frac{20,000 \times 32.2}{1200}} = 23.17 \text{ rad/s}$$

$$\frac{1}{2} m \dot{x}_{\max}^2 = \frac{1}{2} k x_{\max}^2 \quad \dot{x}_{\max} = 23.17 \times 4 = 92.66 \text{ ft/s}$$

$$\begin{aligned} \text{Eq. (2.3-19)} \quad x &= e^{-\omega_n t} [0 + \omega_n x(0)] t + x(0) e^{-\omega_n t} \\ &= e^{-\omega_n t} x(0) [1 + \omega_n t] \end{aligned}$$

$$\frac{x}{12} = e^{-\omega_n t} 4 [1 + \omega_n t] \quad \text{or} \quad e^{-\omega_n t} [1 + \omega_n t] = .0417$$

solve by trial

$\omega_n t$	$e^{-\omega_n t}$	$e^{-\omega_n t} [1 + \omega_n t]$
4.90	.00745	.0439
4.96	.007017	.04182 ← close
4.97	.006947	.04147

$$\therefore \omega_n t = 4.96$$

$$t = \frac{4.96}{23.17} = 0.214 \text{ s}$$

2-44

$$\omega_n = \sqrt{\frac{35000}{4.53}} = 87.89 \text{ rad/s}$$

$$\tau = \frac{2\pi}{87.89} = .0715 \text{ s}$$

$$c_c = 2\sqrt{km} = 797.04$$

$$\zeta = .2197$$

$$\tau_d = \sqrt{1 - \zeta^2} \tau = .0697$$

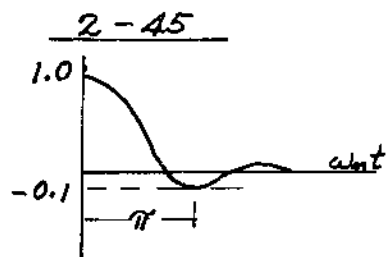
Eq. (2.3-16)

$$x = \frac{\dot{x}(0)}{\omega_n \sqrt{1 - \zeta^2}} e^{-\zeta \omega_n t} \sin \sqrt{1 - \zeta^2} \omega_n t$$

$$\text{at } x_{\max}, \sin \sqrt{1 - \zeta^2} \omega_n t \approx 1.0, \text{ also } \omega_n t = \pi/2$$

$$x = \frac{15.24}{87.89 \times .9756} e^{-.2197(\pi/2)} = .1259 \text{ m}$$

$$t = \frac{1}{4} \tau_d = .0174 \text{ s}$$



Eq. (2.3-16) for $\dot{x}(0) = 0$

$$x = x(0) e^{-\zeta \omega_n t} \cos \sqrt{1-\zeta^2} \omega_n t$$

$$\text{at } \sqrt{1-\zeta^2} \omega_n t = \pi \quad \cos \sqrt{1-\zeta^2} \omega_n t = -1$$

$$-0.1 = 1 e^{-\zeta \omega_n t} (-1) \quad \text{solve by trial}$$

ζ	$\sqrt{1-\zeta^2}$	$\frac{-5\pi}{\sqrt{1-\zeta^2}}$	$e^{\frac{-5\pi}{\sqrt{1-\zeta^2}}}$
.50	.866		.1630
.59	.8074	-2.2957	.1007 ←
.60	.800	-2.3562	.0948

$$\therefore \zeta_1 = 0.59$$

$$\text{If } \zeta = \frac{1}{2} \zeta_1 = 0.295, \quad \sqrt{1-\zeta^2} = .9555$$

$$x_{\text{overshoot}} = 1 e^{\frac{-2.95 \pi}{.9555}} = 0.379 = 37.9\%$$

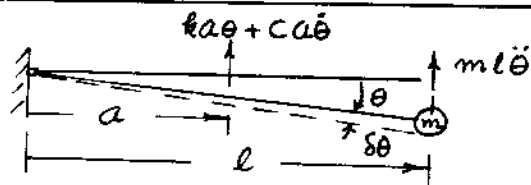
2-46 (Prob. 2-38 by V.W.)

$$\delta W = -m\ddot{\theta} \cdot l \delta\theta - (ka\theta + ca\dot{\theta}) a \delta\theta = 0$$

$$\ddot{\theta} + \frac{ca^2}{ml^2} \dot{\theta} + \frac{ka^2}{ml^2} \theta = 0$$

$$\ddot{\theta} + 2\zeta \dot{\theta} + \omega_n^2 \theta = 0 \quad \therefore \omega_n = \frac{a}{l} \sqrt{\frac{k}{m}}, \quad \zeta = \frac{1}{2} \frac{c}{m} \left(\frac{a}{l}\right)^2 \cdot \frac{l}{a} \sqrt{\frac{m}{k}} = \frac{ca}{2l\sqrt{km}}$$

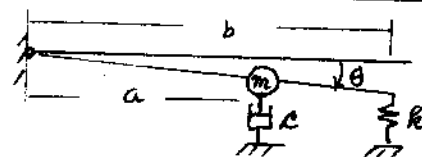
$$\omega_d = \frac{a}{l} \sqrt{\frac{k}{m}} \sqrt{1 - \frac{1}{4} \left(\frac{ca}{2l}\right)^2}$$



2-46 (Prob. 2-39 by V.W.)

$$\delta W = -ma\ddot{\theta} \cdot a\delta\theta - ca\dot{\theta} \cdot a\delta\theta - kb\theta \cdot b\delta\theta = 0$$

$$\therefore \ddot{\theta} + \frac{c}{m} \dot{\theta} + \frac{k}{m} \left(\frac{b}{a}\right)^2 \theta = 0$$

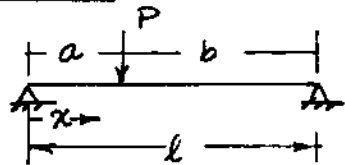


2-47

$$k_{\text{eff}} = (k_1 + k_2) \text{ in series with } k_3$$

$$= \frac{(k_1 + k_2) k_3}{k_1 + k_2 + k_3}$$

2-48



$$y(x) = \frac{Pbx}{6EI} (l^2 - x^2 - b^2)$$

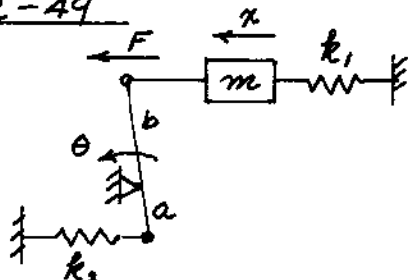
$$0 \leq x \leq a$$

$$\text{Let } x = \frac{l}{3}$$

$$y\left(\frac{l}{3}\right) = \frac{P \frac{2}{3} l \cdot \frac{1}{3} l}{6EI} \left(l^2 - \frac{l^2}{9} - \frac{4}{9} l^2\right) = \frac{Pl^3}{EI} \cdot \frac{4}{243}$$

$$\text{Flexibility} = \frac{y}{P} = \frac{4}{243} \frac{l^3}{EI} \quad \text{at } \frac{x}{l} = \frac{1}{3}$$

2-49



$$F = k_1 b \theta + \frac{a}{b} k_2 a \theta$$

$$x = b \theta$$

$$\therefore F = k_1 x + \left(\frac{a}{b}\right)^2 k_2 x$$

$$k_{\text{eff}} = \frac{F}{x} = k_1 + \left(\frac{a}{b}\right)^2 k_2$$

2-50

$$k_{\text{eff}} = \frac{k_1 k_2}{k_1 + k_2}$$

2-51

Eq. (2.3-16)

$$x(t) = e^{-5\omega_n t} \left(\frac{5}{\sqrt{1-5^2}} \sin \sqrt{1-5^2} \omega_n t + \cos \sqrt{1-5^2} \omega_n t \right)$$

at $\omega_n t = 2\pi, 4\pi, 6\pi, \text{etc.}$

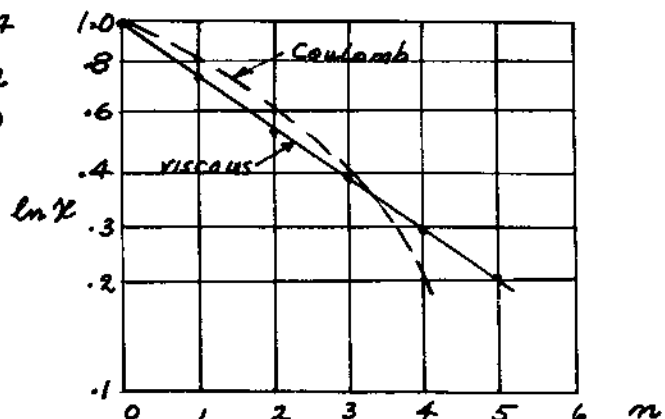
$$x(t) \approx e^{-5\omega_n t} (0 + 1)$$

n	$\omega_n t$	$e^{-.05 a \omega_n t}$	X_m
0	0	1.0	1.0
1	2π	.7304	.8
2	4π	.5335	.6
3	6π	.3896	.4
4	8π	.2845	.2
5	10π	.2078	0

For Coulomb friction

$$X_1 - X_2 = \frac{4 F_d}{k} = \frac{4 \times .05 k}{k} = .20$$

$$X_m = 1 - .2n$$



2-52

$$T = \frac{1}{2} m_1 \dot{x}^2 + \frac{1}{2} m_2 \dot{x}^2 + \frac{1}{2} I_0 \left(\frac{\dot{x}}{r} \right)^2$$

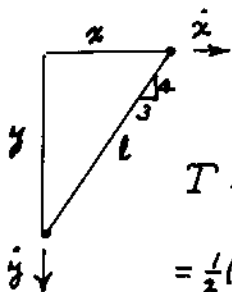
$$U = \frac{1}{2} k_1 x^2 + \frac{1}{2} k_2 \left(\frac{a}{r} x + x \right)^2$$

$$\frac{d}{dt}(T+U) = (m_1 \ddot{x} + m_2 \ddot{x} + \frac{I_0}{r^2} \ddot{x}) \dot{x} + (k_1 x + k_2 x + \frac{a}{r} k_2 x) \dot{x} = -c \dot{x} \dot{x} = \frac{dW}{dt}$$

$$(m_1 + m_2 + I_0/r^2) \ddot{x} + (k_1 + k_2 + \frac{a}{r} k_2) x + c \dot{x} = 0$$

$$c_c = 2 \sqrt{k_{\text{eff}} m_{\text{eff}}} = 2 \sqrt{(k_1 + k_2 + \frac{a}{r} k_2) (m_1 + m_2 + I_0/r^2)}$$

2-53



$$x^2 + y^2 = l^2$$

$$2x dx + 2y dy = 0$$

$$\therefore \frac{dy}{dx} = -\frac{x}{y}$$

$$\dot{y} = -\frac{3}{4} \dot{x}$$

$$T = \frac{1}{2} (ml) \frac{l^2}{3} \left(\frac{\dot{x}}{l} \right)^2 + \frac{1}{2} (ml) \left[\left(\frac{\dot{x}}{2} \right)^2 + \left(\frac{\dot{y}}{2} \right)^2 \right] + \frac{1}{2} (ml) \frac{l^2}{12} \left[\frac{4\dot{x}}{5l} + \frac{3\dot{y}}{5l} \right]^2 + \frac{1}{2} M \dot{y}^2$$

$$= \frac{1}{2} (ml) \left\{ \frac{l^2}{3} \left(\frac{\dot{x}}{l} \right)^2 + \left[\left(\frac{\dot{x}}{2} \right)^2 + \left(\frac{3\dot{x}}{8} \right)^2 \right] + \frac{l^2}{12} \left[\frac{4}{5} \frac{\dot{x}}{l} + \frac{9}{20} \frac{\dot{x}}{l} \right]^2 + \frac{1}{2} M \left(\frac{3\dot{x}}{4} \right)^2 \right\}$$

$$= \frac{1}{2} (ml) \left[\frac{1}{3} + \frac{1}{4} + \frac{9}{64} + \frac{1}{12} \left(\frac{16}{25} + \frac{81}{400} + \frac{72}{100} \right) \right] \dot{x}^2 + \frac{1}{2} M \frac{9}{16} \dot{x}^2$$

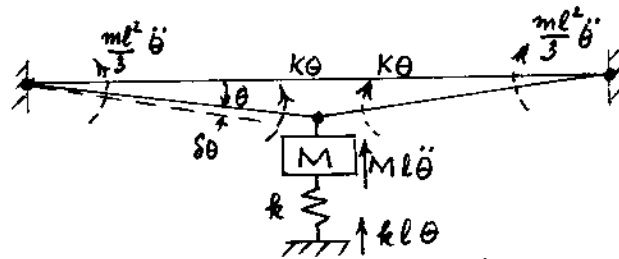
$$= \frac{1}{2} \left[(.854 ml) + .5625 M \right] \dot{x}^2$$

$$U = \frac{1}{2} k y^2 = \frac{1}{2} k \frac{9}{16} x^2$$

$$\frac{d}{dt}(T+U) = -c \frac{2}{3} l \frac{\dot{x}}{l} = -\frac{2}{3} c \dot{x}$$

$$(0.854 ml + .5625 M) \ddot{x} + .5625 k x + \frac{2}{3} c \dot{x} = 0$$

2-54

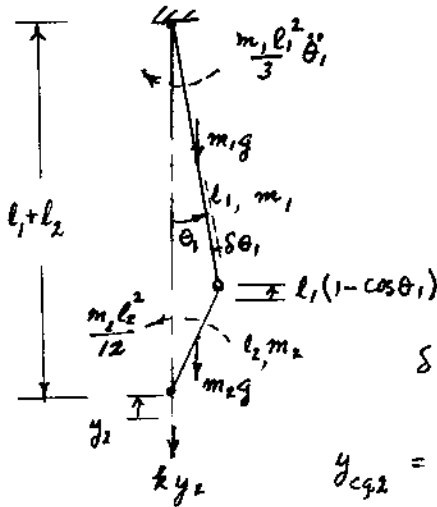


$$\delta W = [-Ml\ddot{\theta} \delta\theta - k\delta\theta l\delta\theta] - 2K\theta \cdot \delta\theta - 2 \frac{ml^2}{3} \ddot{\theta} \delta\theta = 0$$

$$(Ml^2 + \frac{2}{3} ml^2) \ddot{\theta} + (kl^2 + 2K) \theta = 0$$

2-55

Accel of c.g. in vertical direction $\cong 0$
but work done by gravity is not zero



$$\delta y_{cg,1} = \frac{l_1}{2} \sin \theta_1 \delta \theta_1 \cong \frac{l_1}{2} \theta_1 \delta \theta_1$$

$$y_2 = l_1(1 - \cos \theta_1) + l_2(1 - \cos \theta_2)$$

$$l_1 \theta_1 \cong l_2 \theta_2, \quad \theta_2 = \frac{l_1}{l_2} \theta_1, \quad \therefore \delta \theta_2 = \frac{l_1}{l_2} \delta \theta_1$$

$$\delta y_2 = l_1(\theta_1 + \frac{l_1}{l_2} \theta_1) \delta \theta_1 = l(1 + \frac{l_1}{l_2}) \theta_1 \delta \theta_1$$

$$\begin{aligned} y_{cg,2} &= l_1(1 - \cos \theta_1) + \frac{l_2}{2}(1 - \cos \theta_2) \quad \therefore \delta y_{cg,2} = l_1 \theta_1 \delta \theta_1 + \frac{l_2}{2} \theta_2 \delta \theta_2 \\ &= l_1 \theta_1 \delta \theta_1 + \frac{l_2}{2} \frac{l_1}{l_2} \theta_1 \frac{l_1}{l_2} \delta \theta_1 \\ &= l_1(1 + \frac{l_1}{2l_2}) \theta_1 \delta \theta_1 \end{aligned}$$

$$\begin{aligned} \delta W &= -(\frac{m_1 l_1^2}{3}) \ddot{\theta}_1 \delta \theta_1 - (\frac{m_2 l_2^2}{12}) \ddot{\theta}_2 \delta \theta_2 - m_1 g \frac{l_1}{2} \theta_1 \delta \theta_1 - m_2 g l_1(1 + \frac{l_1}{2l_2}) \theta_1 \delta \theta_1 \\ &\quad - k[l_1(1 - \cos \theta_1) + l_2(1 - \cos \theta_2)] l(1 + \frac{l_1}{l_2}) \theta_1 \delta \theta_1 = 0 \end{aligned}$$

$$\text{sub. } \ddot{\theta}_2 = \frac{l_1}{l_2} \ddot{\theta}_1, \quad \delta \theta_2 = \frac{l_1}{l_2} \delta \theta_1, \quad \theta_2 = \frac{l_1}{l_2} \theta_1$$

$$[\frac{m_1 l_1^2}{3} + \frac{m_2 l_2^2}{12} (\frac{l_1}{l_2})^2] \ddot{\theta}_1 + [m_1 g \frac{l_1}{2} + m_2 g l_1(1 + \frac{l_1}{2l_2})] \theta_1 = 0$$

(spring force is 2nd order infinitesimal)

$$\underline{3-1} \quad x_{res.} = \frac{F}{c\omega_n} = \frac{F\gamma}{2\pi c}$$

$$c = \frac{F\gamma}{2\pi x_{res}} = \frac{24.46 \times 0.20}{2\pi \cdot 1.27 \times 10^{-2}} = 61.3 \quad \frac{Ns}{m}$$

3-2

$$\frac{x_{undamped}}{x_{damped}} = \sqrt{\frac{(\omega_n^2 - \omega^2)^2 + (c\omega/m)^2}{(\omega_n^2 - \omega^2)^2}} = R$$

$$\omega_n = \frac{2\pi}{\gamma} = \frac{6.283}{.20} = 31.416 \quad \omega = 8\pi = 25.13$$

$$\frac{c\omega}{m} = \frac{61.3 \times 8\pi}{1.95} = 790.1$$

$$R = \sqrt{\frac{(31.4^2 - 25.13^2)^2 + (790.1)^2}{(31.4^2 + 25.13^2)^2}} = 2.44$$

$$\underline{3-3} \quad \delta = \ln 4.2 = 1.435 = \frac{2\pi\zeta}{\sqrt{1-\zeta^2}}$$

square & solve for ζ^2

$$2.0595(1-\zeta^2) = 39.478\zeta^2$$

$$\zeta^2 = \frac{2.059}{41.538} = .0496, \quad \zeta = .223$$

$$\omega_d = \frac{2\pi}{\tau_d} = \frac{2\pi}{1.80} = \omega_n \sqrt{1-\zeta^2} \quad \omega_n = \frac{2\pi}{1.80\sqrt{1-.0496}} = 3.5806$$

$$\omega = 3 \quad \frac{\omega}{\omega_n} = .8378$$

$$\begin{aligned} \text{Eq(3.1-7)} \quad x &= \frac{F_0/k}{\sqrt{[1-(\frac{\omega}{\omega_n})^2]^2 + [2\zeta\frac{\omega}{\omega_n}]^2}} = \frac{2/525}{\sqrt{[1-.702]^2 + .1396}} \\ &= \frac{2/525}{.4779} = .00797 \text{ m} = .797 \text{ cm} \end{aligned}$$

Eq(3.1-8)

$$\phi = \tan^{-1} \frac{2\zeta(\frac{\omega}{\omega_n})}{1-(\frac{\omega}{\omega_n})^2} = \tan^{-1} \frac{.446 \times .8378}{.29801} = \tan^{-1} 1.2538 = 51.43^\circ$$

3-4 Square Eq. (3.1-7)

$$\left(\frac{X}{X_0}\right)^2 = \frac{1}{(1-r)^2 + 4\zeta^2 r}$$

where $X_0 = F/k$
 $r = (\omega/\omega_n)^2$

$$\frac{\partial}{\partial r} \left(\frac{X}{X_0}\right)^2 = \frac{2(1-r) - 4\zeta^2}{(\text{denom.})^2} = 0 \quad \therefore r = 1 - 2\zeta^2 = (\omega/\omega_n)_p^2$$

$$\left(\frac{\omega}{\omega_n}\right)_p = \sqrt{1 - 2\zeta^2}$$

3-5

At resonance $\frac{\omega}{\omega_n} = 1.0$, $\frac{X}{X_0} = \frac{1}{2\zeta} = \frac{.58}{X_0}$

When $\frac{\omega}{\omega_n} \neq 1.0$

$\therefore X_0 = 1.16\zeta$

$$\frac{X}{X_0} = \frac{1}{\sqrt{[1-r^2]^2 + [2\zeta r]^2}} = \frac{.46}{X_0}$$

Square & solve for ζ^2

$$\frac{1}{[1-.64]^2 + [2\zeta]^2 \cdot .64} = \frac{.2116}{(1.16\zeta)^2}$$

$$6.359 \zeta^2 = .1296 + 2.560 \zeta^2$$

$$\zeta^2 = .0341$$

$$\zeta = .1847$$

3-6

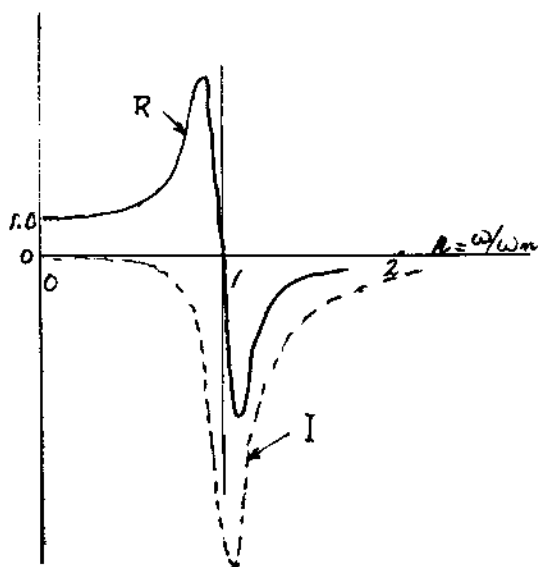
Eq. 3.1-17 $H(\omega) = \frac{1-r^2}{[1-r^2]^2 + [2\zeta r]^2} - i \frac{2\zeta r}{[1-r^2]^2 + [2\zeta r]^2}$
 $= R + i I$

$\zeta = .01$

r	$1-r^2$	$(1-r^2)^2$	$2\zeta r$	$(2\zeta r)^2$	Den.	R	I
.20	.9999	.9998	.0040	.00016	.99982	1.00008	-.004001
.40	.840	.7056	.0080	.00064	.70566	1.19037	-.011337
⋮							
.90	.1900	.0361	.0180	.000324	.036424	5.21634	-.494180
.94	.1164	.013549	.01880	.000353	.013902	8.37263	-1.35228
.96						12.0325	-2.94676

3-6 Cont.

.990	.01990	.000396	.01980	.000392	.000788	25.2525	-25.1256
.998						9.6428	-48.1658
.999						4.9578	-49.5535
1.000						0	-50.000
1.002						-9.5881	-47.988
1.020						-19.725	-9.9602
1.060	-.1236	.015277	.0212	.000449	.015726	-7.8594	-1.348



3-7

$$m\ddot{x}_1 = -c\dot{x}_1 + k(x_2 - x_1)$$

$$m\ddot{x}_1 + c\dot{x}_1 + kx_1 = kX_2 \sin \omega t$$

Replace excitation by $kX_2 e^{i\omega t}$, then $x = X_1 e^{i(\omega t - \phi)}$
 $= X_1 e^{-i\phi} e^{i\omega t} = \bar{X}_1 e^{i\omega t}$

$$[(k - m\omega^2) + i\omega c] \bar{X}_1 e^{i\omega t} = kX_2 e^{i\omega t}$$

$$\bar{X}_1 = \frac{kX_2}{(k - m\omega^2) + i\omega c} = \frac{kX_2 e^{-i\phi}}{\sqrt{(k - m\omega^2)^2 + (c\omega)^2}}$$

$$\therefore X_1 = \frac{kX_2}{\sqrt{(k - m\omega^2)^2 + (c\omega)^2}}, \quad \phi = \tan^{-1} \frac{c\omega}{k - m\omega^2}$$

3-8

$$m\ddot{x} = c(\dot{y} - \dot{x}) + kx$$

$$m\ddot{x} + c\dot{x} + kx = c\dot{y}$$

3-8 Cont.

$$\text{let } y = Y e^{i\omega t}$$

$$\dot{y} = i\omega Y e^{i\omega t}$$

$$x = X e^{i(\omega t - \phi)} = X e^{-i\phi} e^{i\omega t} = \bar{X} e^{i\omega t}$$

$$(k - m\omega^2 + i\omega c) \bar{X} = i\omega Y$$

$$\bar{X} = \frac{i\omega Y}{k - m\omega^2 + i\omega c} = \frac{i\omega Y e^{-i\gamma}}{\sqrt{(k - m\omega^2)^2 + (c\omega)^2}} = X e^{-i\phi}$$

$$X e^{-i\phi} = \frac{\omega Y}{\sqrt{(k - m\omega^2)^2 + (c\omega)^2}} i e^{-\gamma} \quad \therefore e^{-i\phi} = i e^{-i\gamma} = e^{-i(\gamma - \frac{\pi}{2})}$$

$$X = \frac{\omega Y}{\sqrt{(k - m\omega^2)^2 + (c\omega)^2}} \quad \phi = \gamma - \frac{\pi}{2}, \quad \tan \gamma = \frac{c\omega}{k - m\omega^2}$$

3-9

$$(a) \quad \varphi = 90^\circ \therefore \text{resonance} \quad \omega_n = \omega = \frac{900 \times 2\pi}{60} = 2\pi f_n$$

$$f_n = 15 \text{ cps} = 900 \text{ cpm}$$

(b) Eq. (3.2-4) at resonance

$$\zeta = \frac{cm}{2MX} = \frac{2 \times 0.0921}{2 \times 181.4 \times 21.6 \times 10^{-3}} = 0.0118 \times 2$$

$$(c) \quad \text{At } 1200 \text{ rpm} \quad \frac{\omega}{\omega_n} = \frac{1200}{900} = 1.333$$

Eq. (3.2-4)

$$X = \frac{me}{M} \frac{1.333^2 \times 2}{\sqrt{[1 - 1.333^2]^2 + [2 \times 0.0118 \times 1.333]^2}}$$

$$= \frac{2 \times 0.0921}{181.4} \times \frac{1.777}{0.78023} = 0.002314 \text{ m} = 0.2314 \text{ cm}$$

(d)

$$\text{Eq. (3.2-5)} \quad \phi = \tan^{-1} \frac{2 \times 0.0235 \times 1.333}{1 - 1.333^2} = \tan^{-1}(-0.0805)$$

$$= 180^\circ - 4.61^\circ = 175.4^\circ$$

3-10

$$M\ddot{x} + c\dot{x} + kx = (m\omega^2)\sin\omega t$$

$$\text{Let } (m\omega^2)\sin\omega t = F e^{i\omega t}$$

$$\text{then } x = X e^{i(\omega t - \phi)} = \bar{X} e^{-i\phi} e^{i\omega t} = \bar{X} e^{i\omega t}$$

$$(-\omega^2 M + i c \omega + k) \bar{X} e^{i\omega t} = F e^{i\omega t}$$

$$\bar{X} = \frac{F}{(k - \omega^2 M) + i(c\omega)}$$

3-11

$$\omega = \frac{1200}{60} = 20 \text{ rps} = 20 \times 2\pi \text{ rad/s.}$$

$$\omega_n = 18 \text{ cps} = 18 \times 2\pi \text{ rad/s. } \therefore \frac{\omega}{\omega_n} = 1.111$$

Eq. 3.2-6

$$x(t) = \underbrace{X_1 e^{-\zeta \omega_n t} \sin(\sqrt{1-\zeta^2} \omega_n t - \phi_1)}_{\text{transient}} + \underbrace{\frac{m\omega^2 \sin(\omega t - \phi)}{\sqrt{(k - m\omega^2)^2 + (c\omega)^2}}}_{\text{steady state}}$$

$$\text{at } t=0, x(0)=0$$

$$0 = -X_1 \sin \phi_1 + \frac{(m\omega^2/k) \sin(-\phi)}{\sqrt{[1 - (\omega/\omega_n)^2]^2 + [2\zeta \omega/\omega_n]^2}} \quad (1)$$

$$\text{at } t=0, \dot{x}(0)=0$$

$$\dot{x}(t) = X_1 e^{-\zeta \omega_n t} \left[\sqrt{1-\zeta^2} \omega_n \cos(\sqrt{1-\zeta^2} \omega_n t - \phi_1) - \zeta \omega_n \sin(\sqrt{1-\zeta^2} \omega_n t - \phi_1) \right] + \frac{(m\omega^2/k) \omega \cos(\omega t - \phi)}{\sqrt{[1 - (\omega/\omega_n)^2]^2 + [2\zeta \omega/\omega_n]^2}}$$

$$\therefore \dot{x}(0)=0 = X_1 [\omega_n \sqrt{1-\zeta^2} \cos \phi_1 + \zeta \omega_n \sin \phi_1] + \frac{(m\omega^2/k) \omega \cos \phi}{\sqrt{[1 - (\omega/\omega_n)^2]^2 + [2\zeta \omega/\omega_n]^2}} \quad (2)$$

$$\text{from (1)} \quad X_1 \sin \phi_1 = \frac{-(m\omega^2/k) \sin \phi}{\sqrt{[1 - (\omega/\omega_n)^2]^2 + [2\zeta \omega/\omega_n]^2}}$$

from (2)

$$X_1 \omega_n \sqrt{1-\zeta^2} \cos \phi_1 + \zeta \omega_n \left[\frac{-(m\omega^2/k) \sin \phi}{\sqrt{[1 - (\omega/\omega_n)^2]^2 + [2\zeta \omega/\omega_n]^2}} + \frac{(m\omega^2/k) \omega \cos \phi}{\sqrt{[1 - (\omega/\omega_n)^2]^2 + [2\zeta \omega/\omega_n]^2}} \right] = 0$$

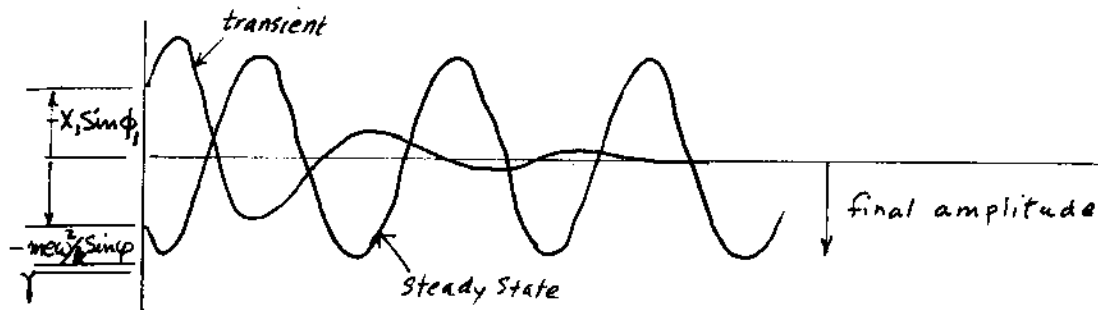
dividing

$$\therefore \tan \phi_1 = \frac{-\sqrt{1-\zeta^2} \sin \phi}{\zeta \sin \phi - \frac{\omega}{\omega_n} \cos \phi} \quad (3)$$

3-11 Cont.

at $t = \infty$, transient term = 0 due to $e^{-5\omega_n t} \rightarrow 0$

$$\therefore \text{final ampl.} = x(\infty) = \frac{(m\omega^2/k) \sin(\omega t - \phi)}{\sqrt{[1 - (\frac{\omega}{\omega_n})^2]^2 + [25 \frac{\omega}{\omega_n}]^2}}$$



$$\begin{aligned} \text{phase } \phi \text{ can then be solved from } \tan \phi &= \frac{25 \frac{\omega}{\omega_n}}{1 - (\frac{\omega}{\omega_n})^2} = \frac{.2(1.111)}{1 - (1.111)^2} \\ &= -.94755 \quad \therefore \phi = -43.457^\circ \quad \text{Eq. (3.2-5)} \end{aligned}$$

Then ϕ_1 solved from (3) $\tan \phi_1 = -.7820$, $\phi_1 \approx 142^\circ$

Then solve for X_1 and sub. into Eq. (3.2-6) for build up eq.

3-12

Lowest critical speed = fundamental freq. of lateral vibr.

$$f = \frac{1}{2\pi} \sqrt{\frac{k}{m}} \quad \text{where } k = \frac{48EI}{l^3}, \quad I = \frac{\pi d^4}{64}$$

$$m = m_{\text{disk}} + 0.486 m_{\text{shaft}}$$

$$k = \frac{48(29 \times 10^6) \pi (\frac{1}{2})^4}{(24)^3 \times 64} = 309.16/\text{in}$$

$$.486 m_{\text{shaft}} = \frac{.486 (.283) \frac{\pi}{4} (\frac{1}{2})^2 (24)}{386} = \frac{.648}{386}$$

$$m = \frac{10}{386} + \frac{.648}{386} = \frac{10.65}{386}$$

$$f = \frac{1}{2\pi} \sqrt{\frac{309 \times 386}{10.65}} = 16.84 \text{ cps} = 1028 \text{ rpm}$$

3-13

$$W = 10 \text{ lb} = 44.48 \text{ N} \quad m = \frac{44.48}{9.81} = 4.534 \text{ kg}$$

$$g = 386 \frac{\text{in}}{\text{sec}^2} = 9.81 \text{ m/s}^2$$

$$k = \frac{48EI}{l^3} = \begin{cases} E = 200 \times 10^9 \text{ N/m}^2 \\ l = 2 \times .3048 = .6096 \text{ m} \\ d = .5 \times 2.54 \times 10^{-2} = 1.270 \times 10^{-2} \text{ m} \\ I = \frac{\pi d^4}{64} = .1277 \times 10^{-8} \end{cases}$$

$$k = \frac{200 \times 10^9 \times 48 \times .1277 \times 10^{-8}}{(.6096)^3} = 54116 \text{ N/m}$$

$$.486 m_{\text{shaft}} = .486 \times \frac{\pi}{4} (1.270 \times 10^{-2})^2 (.6096) \times \rho = .2938$$

$$\rho = 7830 \frac{\text{kg}}{\text{m}^3} = \text{density of steel}$$

$$f = \frac{1}{2\pi} \sqrt{\frac{54116}{4.534 + .2938}} = 16.86 \text{ Hz}$$

3-14

$$\text{dia} = 2.54 \text{ cm}$$

$$I = \frac{\pi d^4}{64} = \frac{\pi \times 41.62}{64 \times 100^4} = 2.043 \times 10^{-8} \text{ m}^4$$

$$k = \frac{48EI}{l^3} = \frac{48(200 \times 10^9) 2.043 \times 10^{-8}}{.4064^3} = 2.922 \times 10^6 \text{ N/m}$$

$$m = 13.6 + .486 \left(\frac{\pi}{4} \times .0254^2 \times .4064 \right) (7830) = 13.6 + .784 = 14.38 \text{ kg}$$

$$f_n = \frac{1}{2\pi} \sqrt{\frac{2.922 \times 10^6}{14.38}} = 71.74 \text{ Hz} = 4304 \text{ rpm}$$

$$\frac{\omega}{\omega_n} = \frac{6000}{4304} = 1.394 \quad r = \frac{e \left(\frac{\omega}{\omega_n} \right)^2}{1 - \left(\frac{\omega}{\omega_n} \right)^2} = -2.060 e$$

$$me = .2879 \text{ kg cm (given)} \quad \therefore e = \frac{.2879}{13.6} = .02117 \text{ cm}$$

$$r = -2.060(.02117) = -.04316 \text{ cm}$$

$$F = m(r+e)\omega^2 = 14.38 \left(\frac{-.04316 + .02117}{100} \right) (2\pi \times 100)^2 = 1273 \text{ N}$$

3-14 Cont.

$$\text{for diam.} = 1.905 \text{ cm}$$

$$.486 m_{\text{shaft}} = .486 \left(\frac{\pi}{4} \times .01905^2 \times .4064 \right) (7830) = .4408$$

$$m = m_{\text{disk}} + .483 m_{\text{shaft}} = 14.04 \text{ kg.}$$

$$\bar{I} = \frac{\pi}{64} 1.905^4 \times 100^{-4} = .6464 \times 10^{-8}$$

$$k = \frac{48 (200 \times 10^9) \cdot 6464 \times 10^{-8}}{.4064^3} = 0.9441 \times 10^6$$

$$f_m = 41.27 \text{ Hz} = 2476 \text{ rpm} \quad \frac{\omega}{\omega_m} = \frac{6000}{2476} = 2.423$$

$$r = \frac{.02117 (2.423)^2}{1 - (2.423)^2} = -.02552 \quad r+e = .00435$$

$$F = 14.042 \left(\frac{.00435}{100} \right) \left(2\pi \times \frac{6000}{60} \right)^2 = 241.1 \text{ N}$$

3-15

$$r = r_0 + \frac{e\omega t}{2} \quad (\text{see Ex. 3.4-1})$$

$$.0508 = 0 + .0212 (2\pi \times 100) \frac{t}{2}$$

$$t = \frac{.0508}{6.6602} = .0075 \text{ sec}$$

3-16

$$m\ddot{x} = -k(x-y) \quad \text{where } x = \text{displ. of } m \text{ measured from static equilib. position of } m \text{ with } y = 0$$

$$\text{Let } y = Y \sin \frac{2\pi Vt}{L}$$

$$\text{then } m\ddot{x} + kx = kY \sin \frac{2\pi Vt}{L} = kY \sin \omega t$$

$$\text{where } \omega = \frac{2\pi V}{L} \quad \text{Sol is } x = X \sin \omega t$$

$$X = \frac{Y}{1 - (\omega/\omega_n)^2} \quad \omega_n = \sqrt{\frac{k}{m}}$$

The most unfavorable speed corresponds to $\frac{\omega}{\omega_n} = 1$

$$\therefore V = \frac{L}{2\pi} \sqrt{\frac{k}{m}}$$

3-17 (Ref. Prob 3-16)

$$\text{Eq. (2.2-9)} \quad f_m = \frac{15.76}{\sqrt{\Delta_{mm}}} = \frac{15.76}{\sqrt{101.6}} = 1.563 \text{ Hz}$$

$$\omega_m = 2\pi f_m = 9.824 \text{ rad/s}$$

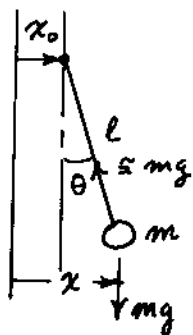
$$V_{\text{crit.}} = \frac{L \omega_m}{2\pi} = \frac{14.63}{2\pi} \times 9.82 = 22.87 \text{ m/s}$$

$$V = 64.4 \frac{\text{km}}{\text{hr}} = \frac{64400}{60^2} = 17.89 \text{ m/s}$$

$$\omega = \frac{2\pi V}{L} = \frac{2\pi \cdot 17.89}{14.63} = 7.683 \text{ rad/s}$$

$$\left(\frac{\omega}{\omega_m}\right)^2 = \left(\frac{7.683}{9.824}\right)^2 = 0.6117 \quad X = \frac{7.62 \text{ cm}}{1 - 0.6117} = 19.62 \text{ cm}$$

3-18



$$x \approx x_0 + l\theta$$

$$\theta = \frac{x - x_0}{l}$$

$$m\ddot{x} = -mg\theta = -\frac{mg}{l}(x - x_0)$$

$$\ddot{x} + \frac{g}{l}x = \frac{g}{l}x_0$$

$$\text{Let } x_0 = X_0 \sin \omega t$$

$$x = X \sin \omega t$$

$$\therefore X = \frac{X_0}{1 - (\omega/\omega_m)^2}$$

$$\omega_m = \sqrt{\frac{g}{l}}$$

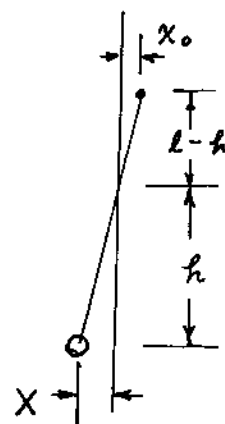
$$\text{when } \omega = \sqrt{2} \omega_m \quad X = -X_0$$

$$\therefore \text{mode found at } \frac{l}{2}$$

$$\frac{h}{|X|} = \frac{l-h}{|X_0|}$$

$$h \left| \frac{X_0}{X} \right| = l-h = h \left[\left(\frac{\omega}{\omega_m} \right)^2 - 1 \right] \quad \text{since } \frac{\omega}{\omega_m} > 1$$

$$h = l \left(\frac{\omega_m}{\omega} \right)^2$$



3-19 $m\ddot{x} + c\dot{x} + kx = c\dot{y} + ky$

$$y = Y \sin \omega t$$

$$x = X \sin(\omega t - \gamma)$$

$$\dot{x} = \omega X \cos(\omega t - \gamma)$$

$$\dot{y} = \omega Y \cos \omega t$$

$$\ddot{x} = -\omega^2 X \sin(\omega t - \gamma)$$

Sub. into DE.

$$(k - m\omega^2)X \sin(\omega t - \gamma) + c\omega X \cos(\omega t - \gamma) = c\omega Y \cos \omega t + kY \sin \omega t$$

Expand $\sin(\omega t - \gamma)$ & $\cos(\omega t - \gamma)$ & equate coef of $\cos \omega t$ & $\sin \omega t$

$$\text{coef. of } \sin \omega t \rightarrow [(k - m\omega^2)\cos \gamma + \omega c \sin \gamma]X = kY$$

$$\text{" " } \cos \omega t \rightarrow [(k - m\omega^2)\sin \gamma - \omega c \cos \gamma]X = -\omega c Y$$

Divide & factor $\cos \gamma$ from num. & denom. to get

$$\tan \gamma = \frac{m c \omega^3}{k(k - m\omega^2) + (c\omega)^2} = \text{Eq. (3.5-9)}$$

Solve for $\sin \gamma$ & $\cos \gamma$ & sub. into

$$\frac{X}{Y} = \frac{k}{(k - m\omega^2)\cos \gamma + \omega c \sin \gamma} = \sqrt{\frac{k^2 + (c\omega)^2}{(k - m\omega^2)^2 + (c\omega)^2}} = \text{Eq. (3.5-8)}$$

after much algebra

3-20

See Sec. (3.6)

$$f = 15.76 \sqrt{\frac{1}{\Delta} \left(\frac{1}{TR} - 1 \right)}$$

$$\frac{1600}{60} = 15.76 \sqrt{\frac{1}{\Delta_{mm}} \left(\frac{1}{.15} - 1 \right)}$$

Solve for $\Delta = 2.678 \text{ mm.}$

For $f = 2200 \text{ cpm}$ the F_{TR} is smaller

3-21

$$TR = \frac{1}{\left(\frac{\omega}{\omega_n}\right)^2 - 1} = 0.10 \quad \therefore \left(\frac{\omega}{\omega_n}\right)^2 = 11.0$$

$$\omega_n^2 = \frac{k}{m} = \frac{\omega^2}{11.0} \quad k = \frac{m\omega^2}{11} = \frac{65}{386} \left(\frac{580 \times 2\pi}{60} \right)^2 \frac{1}{11} = 56.5 \text{ lb/in}$$

$$k_{\text{per spring}} = \frac{1}{3} \times 56.5 = 18.8 \text{ lb/in}$$

3-22

$$k = \frac{Mg}{\Delta} = \frac{453.4 \times 9.81}{.005080} = 875561 \frac{N}{m} = 8755.6 \frac{N}{cm}$$

$$\omega_n^2 = \frac{k}{M} = \frac{8755.6 \times 10^2}{453.4} = 1931.1$$

$$\left(\frac{\omega}{\omega_n}\right)^2 = \left(\frac{1200 \times 2\pi}{60}\right)^2 \frac{1}{1931.1} = 8.177$$

$$X = \frac{\frac{me}{M} \left(\frac{\omega}{\omega_n}\right)^2}{\left(\frac{\omega}{\omega_n}\right)^2 - 1} = \frac{\frac{.2303}{453.4} \times 8.177}{8.177 - 1} = 0.000579 \text{ m}$$

$$F_{TR} = kX = 875561 \times .000579 = 506.7 \text{ N}$$

3-23

$$\text{New } M = 453.4 + 1136. = 1589.4 \text{ kg}$$

$$\text{New } k = 875561 \times \frac{1589.4}{453.4} = 3069.295 \times 10^3 \frac{N}{m}$$

$\frac{\omega}{\omega_n}$ is same

$$X = \frac{\frac{me}{M} \left(\frac{\omega}{\omega_n}\right)^2}{\left(\frac{\omega}{\omega_n}\right)^2 - 1} = \frac{453.4}{1589.4} \times .000579 = 0.165 \times 10^{-3} \text{ m}$$

3-24

$$M = 68 + 1200 = 1268 \text{ kg.}$$

$$f_n = 160 \text{ cpm} \quad \omega_n = \frac{160}{60} \times 2\pi = 16.75 \text{ r/s}$$

$$\frac{\omega}{\omega_n} = \frac{31.4}{16.75} = 1.8746 \quad k = \omega_n^2 M = 355951 \frac{N}{m}$$

$$X = \frac{100 / 0.3559 \times 10^6}{\sqrt{[1 - 1.875^2]^2 + [0.2 \times 1.875]^2}} = 110.5 \times 10^{-6} \text{ m}$$

$$= 0.01105 \text{ cm}$$

$$F_{TR} = kX \sqrt{1 + (2\zeta \frac{\omega}{\omega_n})^2} = .3559 \times 10^6 \times 110.5 \times 10^{-6} \sqrt{1 + .1406}$$

$$= 42.0 \text{ N}$$

3-25

$$\omega_m = \sqrt{\frac{k}{m}} = \sqrt{\frac{280200}{113}} = 49.796$$

$$\omega = 2\pi \times 20 = 125.6 \quad \frac{\omega}{\omega_m} = 2.5236$$

$$\text{Accel.} = 15.24 = \omega^2 Y \quad \therefore Y = .000965 \text{ cm}$$

$$\left| \frac{X}{Y} \right| = \frac{\sqrt{1 + [0.2 \times 2.5236]^2}}{\sqrt{[1 - 6.368]^2 + .2547}} = 0.2078$$

$$\therefore X = .2078 \times .000965 = 0.0002005 \text{ cm}$$

$$\omega^2 X = 3.166 \text{ cm/s}^2 = \text{transmitted accel.}$$

3-26 Add mass M to instrument \therefore increase ω/ω_m

$$X \text{ must be reduced to } \left(\frac{2.03}{3.166} \right) .0002005 = .0001285$$

$$\frac{X}{Y} = \frac{1285}{9650} = .1332 \leq \frac{\sqrt{1 + (.2 \frac{\omega}{\omega_m})^2}}{\sqrt{[1 - (\frac{\omega}{\omega_m})^2]^2 + (.2 \frac{\omega}{\omega_m})^2}}$$

$$\text{Solve by trial: for } \frac{\omega}{\omega_m} = 3, \quad \sqrt{\quad} = .1321$$

$$\text{for } \frac{\omega}{\omega_m} = 4 \quad \sqrt{\quad} = .0893$$

both of these are OK

$$\text{For } \frac{\omega}{\omega_m} = 4 \quad \omega_m = 31.4 = \sqrt{\frac{k}{M_i + M_o}} = \sqrt{\frac{280200}{M_i + M_o}}$$

$$M_i + M_o = 284. \quad \therefore M_o = 171, \text{ kg to be added.}$$

3-27 Compare Eq.(3.5-8) with Eq(3.6-2). They are same

$$TR = \sqrt{\frac{1 + (2\zeta \frac{\omega}{\omega_n})^2}{[1 - (\frac{\omega}{\omega_n})^2]^2 + [2\zeta \frac{\omega}{\omega_n}]^2}}$$

To calculate $20 \log |TR|$, first calculate TR with ζ fixed, varying $\frac{\omega}{\omega_n}$. Then find Db. Suggest computer program.

3-28 $W_d = \pi c \omega X^2 = \pi 2\zeta \frac{\omega}{\omega_n} k X^2$

$$= \frac{\pi F_0^2}{k} \frac{2\zeta (\frac{\omega}{\omega_n})}{[1 - (\frac{\omega}{\omega_n})^2]^2 + [2\zeta \frac{\omega}{\omega_n}]^2}$$

3-29 $\eta = \frac{W_d}{2\pi U}$, $U = \frac{1}{2} k X_{max}^2$, $W_d = \pi c \omega X_{max}^2$

$$\therefore \eta = \frac{\pi c \omega X^2}{2\pi \frac{1}{2} k X^2} = \frac{c \omega}{k}$$

3-30 $\ddot{x} + 2\zeta \omega_n \dot{x} + \omega_n^2 x = \frac{F}{m} \sin \omega t$

$$X_{res.} = \frac{F}{2\zeta k}$$

From Prob. 3-29 $\eta_{res} = \frac{c \omega}{k} = \frac{c \omega_n}{k}$ at resonan.
 $= \frac{c}{c_c} \frac{2m \omega_n^2}{k} = 2\zeta$

$$\therefore \ddot{x} + \eta_{res.} \omega_n \dot{x} + \omega_n^2 x = \frac{F}{m} \sin \omega t$$

3-31 $\frac{\gamma_d}{\gamma_n} = \frac{\omega_n}{\omega_d} = \frac{1}{\sqrt{1-\zeta^2}}$ $\therefore (\frac{\gamma_n}{\gamma_d})^2 = 1 - \zeta^2$

$$(\frac{\gamma_n}{\gamma_d})^2 + \zeta^2 = 1 = \text{circle of radius 1}$$

3-32 $\frac{W_d}{U} = \frac{2\zeta \pi k X^2}{\frac{1}{2} k X^2} = 4\zeta \pi$

but $\delta \cong 2\pi\zeta$ $\therefore \frac{W_d}{U} \cong 2\delta$, $\zeta = \frac{c}{2m \omega_n}$

$$\delta = \frac{c \pi}{m \omega_n} = \frac{c \pi \omega_n}{k} = \frac{W_d}{2U}$$

3-33

$$W_d = f(X, \omega) = \pi c \omega X^2$$

$$\delta = \frac{W_d}{2U} = \frac{\pi c \omega X^2}{k X^2} = \frac{\pi c \omega}{k} \quad \text{for viscous damping}$$

3-34

$$W_d = c_{eq} \pi \omega X^2 = D 4X \quad \text{for Coulomb}$$

$$\therefore c_{eq} = \frac{4D}{\pi \omega X}$$

$$\zeta_{eq} = \frac{c_{eq}}{C_c} = \frac{4D}{\pi \omega X 2m \omega_m}$$

$$2\zeta_{eq} \frac{\omega}{\omega_m} = \frac{4D}{\pi \omega X 2m \omega_m} \cdot \frac{2\omega}{\omega_m} = \frac{4D}{\pi k X}$$

3-35

$$X = \frac{F_0/k}{\sqrt{[1 - (\frac{\omega}{\omega_m})^2]^2 + [2\zeta_{eq} \frac{\omega}{\omega_m}]^2}}$$

$$2\zeta_{eq} \frac{\omega}{\omega_m} = \frac{4D}{\pi k X}$$

Square both sides and subst for ζ_{eq} .

$$X^2 \left\{ [1 - (\frac{\omega}{\omega_m})^2]^2 + [\frac{4D}{\pi k X}]^2 \right\} = (F_0/k)^2$$

$$X^2 [1 - (\frac{\omega}{\omega_m})^2]^2 = (F_0/k)^2 - (\frac{4D}{\pi k})^2$$

$$X = \frac{\sqrt{(F_0/k)^2 - (4D/\pi k)^2}}{1 - (\frac{\omega}{\omega_m})^2} \quad \therefore \text{for } X \text{ real} \quad F_0 > \frac{4D}{\pi}$$

3-36

Rewrite eq. as $\frac{kX}{F_0} = \frac{\sqrt{1 - (\frac{4D}{\pi F_0})^2}}{1 - (\frac{\omega}{\omega_m})^2} = f\left(\frac{D}{F_0}, \frac{\omega}{\omega_m}\right)$

3-37

$$J\ddot{\theta}_2 + K(\theta_2 - \theta_1) = 0 \quad \omega_m^2 = \frac{K}{J}$$

$$\ddot{\theta}_2 + \omega_m^2 \theta_2 = \omega_m^2 \theta_1 = \omega_m^2 \theta_1 \sin \omega t \quad \text{Let } \theta_2 = \Theta_2 \sin \omega t$$

$$\therefore \Theta_2 = \frac{\omega_m^2 \Theta_1}{\omega_m^2 - \omega^2} \quad \text{and} \quad \theta_2 - \theta_1 = (\Theta_2 - \Theta_1) \sin \omega t \quad \begin{cases} \theta_2 = \text{outer wheel} \\ \theta_1 = \text{shaft.} \end{cases}$$

$$\begin{aligned} \text{Rel. ampl} &= (\Theta_2 - \Theta_1) = \frac{\omega_m^2 \Theta_1}{\omega_m^2 - \omega^2} - \Theta_1 = \frac{\omega^2}{\omega_m^2 - \omega^2} \Theta_1 \\ &= \frac{(\frac{\omega}{\omega_m})^2}{1 - (\frac{\omega}{\omega_m})^2} \Theta_1 \end{aligned}$$

$$\text{Rel to fixed ref} \quad \theta_2 = \frac{1}{1 - (\frac{\omega}{\omega_m})^2} \Theta_1$$

3-38

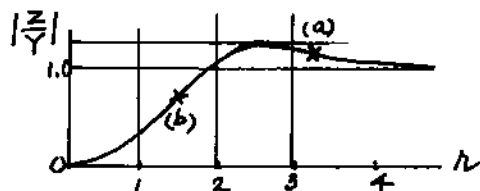
Let $n = \omega/\omega_m$

$$\frac{Z}{Y} = \frac{n^2}{\gamma(1-n^2)^2 + (2\zeta n)^2} \quad \left(\frac{Z}{Y}\right)^2 = \frac{(n^2)^2}{(1-n^2)^2 + (2\zeta n)^2}$$

$$\frac{\partial}{\partial n^2} \left(\frac{Z}{Y}\right) = 0 \text{ gives } n_p^2 = \frac{1}{1-2\zeta^2} \text{ for peak ampl.}$$

$$\text{for } \zeta = 0.650, \quad n_p = 2.54, \quad \frac{Z}{Y} = 1.012$$

$$\text{for (a)} \quad \frac{Z}{Y} = 1.010 \quad \text{Write above eq. as } n^4[1 - (\frac{Y}{Z})^2] - n^2(2-4\zeta^2) + 1 = 0$$



For accuracy expand $1 - (\frac{Y}{Z})^2$

$$= 1 - \frac{1}{(\frac{Z}{Y})^2} = 1 - \frac{1}{(1+0.01)^2}$$

$$= 1 - (1+0.01)^{-2} = 1 - (1 - 0.02 + 0.0003 - \dots)$$

$$= 0.0197 \quad (2-4\zeta^2) = 0.310$$

$$0.0197 n^4 - 0.310 n^2 + 1 = 0$$

$$\frac{\omega}{\omega_m} = n = \begin{cases} 2.125 \\ 3.350 \end{cases}$$

\therefore Lowest freq. for 1% accuracy = $3.35 \times 4.75 = \underline{15.9 \text{ cps.}}$

(b) For 2% above plot indicates we must use $\frac{Z}{Y} = 0.98$

$$-0.0412 n^4 - 0.310 n^2 + 1 = 0, \quad n = 1.57$$

$$1.57 \times 4.75 = \underline{7.45 \text{ cps}}$$

3-39

$$Z = \frac{(\frac{\omega}{\omega_m})^2 Y}{1 - (\frac{\omega}{\omega_m})^2}$$

$$Y = Z \left[\left(\frac{\omega_m}{\omega}\right)^2 - 1 \right]$$

$$= 0.052 \left[\left(\frac{1}{4}\right)^2 - 1 \right] = -0.0488 \text{ cm}$$

3-40

sensitivity = 0.096 rms volts/cm per sec.

0.024 rms volts measured \therefore cm per sec vel = $\frac{0.024}{0.096} = 0.250$

$$\omega = 2\pi \times 30 = 188.49 \text{ rad/s.}$$

$$(a) \text{ vel} = 0.250 = \omega x = 188.49 x_{\text{cm.}} \quad x = \underline{0.001326 \text{ rms cm.}}$$

(b) ampl = 0.40 cm peak

cannot be used since 0.40 cm exceeds clearance of 0.30 cm

3-41 sensitivity = 40 mV/cm per s.

1 g accel = 9.81 m/s² = 981 cm/s² accel = $\omega^2 Y$

(a) at 10 cps $\omega = 2\pi 10 = 62.83$ rad/s vel = ωY .

$$\omega Y = \frac{981}{62.83} = 15.61 \text{ cm/s.}$$

$$\text{Output volts} = 40 \times 15.61 = \underline{\underline{624.5 \text{ mV}}}$$

(b) at 2000 cps $\omega = 12566$

$$\omega Y = \frac{981}{12566} = 0.07806 \text{ cm/s.}$$

$$\text{Output volts} = 40 \times 0.07806 = \underline{\underline{3.123 \text{ mV}}}$$

3-42 For vel. pick-up $\frac{\omega}{\omega_n} \gg 1 \therefore Z = Y$

$$\text{Volts generated by instrument} = \omega Z = \omega Y$$

3-43 Sensitivity = 30 mV/cm. per s.

$$3 \text{ mV} = \text{accuracy limit} = 30 (\omega Z) \therefore \omega Z = 0.10 \text{ cm/s} \\ = \text{Limiting vel.}$$

$$v_{\min} = 0.10 = \frac{981}{2\pi f} \therefore f = \underline{\underline{1561 \text{ cps}}} = \text{upper freq. limit.}$$

$$\text{At } f = 200 \quad v = \frac{981}{2\pi \times 200} = 0.7807$$

$$\text{instr. reading} = 30 \times 0.7807 = \underline{\underline{23.42 \text{ mV.}}}$$

3-44 $C = 450 \text{ pF}$ $Q = 18 \text{ pC/g}$

$$C_{\text{cable}} = 5 \times 50 = 250 \text{ pF}$$

$$E = \frac{Q}{C} = \frac{18}{450} = 0.040 \text{ volts/g} = \underline{\underline{40 \text{ mV/g open circuit}}}$$

With cable

$$E = 40 \times \frac{450}{450 + 250} = \underline{\underline{25.7 \text{ mV/g}}}$$

3-45

$$\frac{W_d}{U} = \frac{\pi C \omega X^2}{\frac{1}{2} k X^2} = \frac{2\pi C \omega}{k}$$

$$C = \zeta C_c = \zeta 2 \sqrt{k m}$$

$$\frac{W_d}{U} = 2\pi \cdot 2\zeta \sqrt{k m} \cdot \frac{\omega}{k} = 4\pi \zeta \frac{\omega}{\omega_m}$$

3-46

$$\frac{W_d}{U} = 4\pi \zeta \frac{\omega}{\omega_m}, \quad \zeta \approx \frac{\delta}{2\pi}$$

$$= 2\delta \frac{\omega}{\omega_m}$$

3-47

$$\eta = \frac{W_d}{2\pi U} = \frac{1}{2\pi} (4\pi \zeta \frac{\omega}{\omega_m}) = 2\zeta \frac{\omega}{\omega_m}$$

but $2\zeta = \gamma$ at resonance

$$\therefore \eta = \gamma \frac{\omega}{\omega_m} = \gamma \quad \text{at resonance}$$

3-48

$$x = \frac{(1-r^2)}{(1-r^2)^2 + (2\zeta r)^2}, \quad y = \frac{-2\zeta r}{(1-r^2)^2 + (2\zeta r)^2}$$

$$y + \frac{1}{2(2\zeta r)} = \frac{-2\zeta r}{(1-r^2)^2 + (2\zeta r)^2} + \frac{1}{2(2\zeta r)} = \frac{-2(2\zeta r)^2 + (1-r^2)^2 + (2\zeta r)^2}{2(2\zeta r)[(1-r^2)^2 + (2\zeta r)^2]}$$

$$= \frac{(1-r^2)^2 - (2\zeta r)^2}{2(2\zeta r)[(1-r^2)^2 + (2\zeta r)^2]}$$

$$\therefore x^2 + \left(y + \frac{1}{2(2\zeta r)}\right)^2 = \frac{(1-r^2)^2}{[(1-r^2)^2 + (2\zeta r)^2]} + \left\{ \frac{(1-r^2)^2 - (2\zeta r)^2}{2(2\zeta r)[(1-r^2)^2 + (2\zeta r)^2]} \right\}^2$$

$$= \frac{1}{4(2\zeta r)^2}$$

4-1

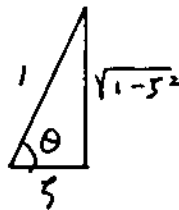
Eq. (4.1-6) for impulsive response

$$x = \frac{\hat{F}}{m\omega_n \sqrt{1-\zeta^2}} e^{-\zeta\omega_n t} \sin \sqrt{1-\zeta^2} \omega_n t$$

For max response $\frac{dx}{dt} = 0$

$$\frac{dx}{dt} = \frac{\hat{F}}{m\omega_n \sqrt{1-\zeta^2}} e^{-\zeta\omega_n t} \left\{ -\zeta\omega_n \sin \sqrt{1-\zeta^2} \omega_n t + \sqrt{1-\zeta^2} \omega_n \cos \sqrt{1-\zeta^2} \omega_n t \right\} = 0$$

$$\therefore \tan \sqrt{1-\zeta^2} \omega_n t = \frac{\sqrt{1-\zeta^2}}{\zeta}$$

4-2

$$\text{Let } \theta = \sqrt{1-\zeta^2} \omega_n t$$

$$\tan \theta = \frac{\sqrt{1-\zeta^2}}{\zeta} \quad \text{from Prob 4-1}$$

$$\sin \theta = \sqrt{1-\zeta^2}$$

$$\text{Eq. (4.1-6) becomes } x_{\text{peak}} = \frac{\hat{F}}{m\omega_n \sqrt{1-\zeta^2}} e^{-\frac{\zeta}{\sqrt{1-\zeta^2}} \tan^{-1} \frac{\sqrt{1-\zeta^2}}{\zeta}} \times \frac{1}{\sqrt{1-\zeta^2}}$$

$$\frac{x_{\text{peak}} \sqrt{k/m}}{\hat{F}} = \exp \left(-\frac{\zeta}{\sqrt{1-\zeta^2}} \tan^{-1} \frac{\sqrt{1-\zeta^2}}{\zeta} \right)$$

4-3

From Ex 4.2-1

$$\frac{xk}{F_0} = 1 - \frac{e^{-\zeta\omega_n t}}{\sqrt{1-\zeta^2}} \cos(\sqrt{1-\zeta^2} \omega_n t - \gamma)$$

$$\tan \gamma = \frac{\zeta}{\sqrt{1-\zeta^2}}$$

$$\text{For peak response } \frac{d}{d(\omega_n t)} \left(\frac{xk}{F_0} \right) = 0$$

$$\left\{ -\zeta \cos(\sqrt{1-\zeta^2} \omega_n t - \gamma) - \sqrt{1-\zeta^2} \sin(\sqrt{1-\zeta^2} \omega_n t - \gamma) \right\} = 0$$

$$\therefore \tan(\sqrt{1-\zeta^2} \omega_n t - \gamma) = \frac{-\zeta}{\sqrt{1-\zeta^2}} \quad \text{expand}$$

$$\frac{\tan \sqrt{1-\zeta^2} \omega_n t - \tan \gamma}{1 + \tan \sqrt{1-\zeta^2} \omega_n t \cdot \tan \gamma} = \frac{-\zeta}{\sqrt{1-\zeta^2}}$$

$$\tan \sqrt{1-\zeta^2} \omega_n t - \frac{\zeta}{\sqrt{1-\zeta^2}} = -\frac{\zeta}{\sqrt{1-\zeta^2}} \left[1 + \frac{\zeta}{\sqrt{1-\zeta^2}} \tan \sqrt{1-\zeta^2} \omega_n t \right]$$

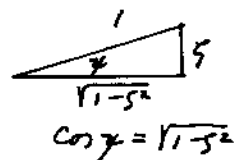
$$\tan \sqrt{1-\zeta^2} \omega_n t \left[1 + \frac{\zeta^2}{1-\zeta^2} \right] = \tan \sqrt{1-\zeta^2} \omega_n t = 0 \quad \therefore \omega_n t = \frac{\pi}{\sqrt{1-\zeta^2}}$$

4-4 Subst. $\omega_n t = \frac{\pi}{\sqrt{1-\zeta^2}}$ into Eq. for peak $\frac{xk}{F_0}$

$$\left(\frac{xk}{F_0}\right)_{\text{peak}} = 1 - \frac{1}{\sqrt{1-\zeta^2}} \exp\left(\frac{-5\pi}{\sqrt{1-\zeta^2}}\right) \cos(\pi - \gamma)$$

$$= 1 + \frac{1}{\sqrt{1-\zeta^2}} \exp\left(\frac{-5\pi}{\sqrt{1-\zeta^2}}\right) \cos \gamma$$

$$\left(\frac{xk}{F_0}\right)_{\text{peak}} = 1 + \exp\left(\frac{-5\pi}{\sqrt{1-\zeta^2}}\right)$$



4-5 From Ex. 4.2-1 $x = \frac{F_0}{k} (1 - \cos \omega_n t)$ $t \geq 0$

For neg. step of $2F_0$ applied at $t = t_0$, we have

$$x = -\frac{2F_0}{k} [1 - \cos \omega_n (t - t_0)] \quad t \geq t_0$$

Adding

$$x = \frac{F_0}{k} \{ \cos \omega_n (t - t_0) - \cos \omega_n t \} \quad t \geq t_0$$

4-6 With zero initial cond. Eq. 4.2-1 gives

$$x(t) = \int_0^t f(\xi) \sin \omega_n (t - \xi) d\xi = \text{particular integral}$$

Solution for homogeneous eq. is from Eq. 2.2-6

$$x(t) = \frac{\dot{x}(0)}{\omega_n} \sin \omega_n t + x(0) \cos \omega_n t$$

Complete sol. is sum of these solutions

4-7 From Eq. 4.2-1 response to unit step function is

$$g(t) = \int_0^t h(t - \xi) d\xi$$

Differentiate both sides w.r.t. time

$$\dot{g}(t) = h(t)$$

4-8 Eq. 4.2-1 $x = \int_0^t f(\xi) h(t-\xi) d\xi$

From Prob. 4-7 $\dot{g}(t-\xi) = h(t-\xi)$

$x(t) = \int_0^t f(\xi) \dot{g}(t-\xi) d\xi$ Integrate by parts

$$x(t) = -f(\xi)g(t-\xi) \Big|_0^t + \int_0^t \dot{f}(\xi)g(t-\xi) d\xi$$

$$= -f(t)g(0) + f(0)g(t) + \int_0^t \dot{f}(\xi)g(t-\xi) d\xi \quad \text{but } g(0) = 0$$

$$\therefore x(t) = f(0)g(t) + \int_0^t \dot{f}(\xi)g(t-\xi) d\xi$$

4-9 $\bar{x}(s) = \frac{(ms+c)x(0) + m\dot{x}(0)}{ms^2 + cs + k}$

$$= \frac{(s+2.5\omega_n)x(0)}{s^2 + 2.5\omega_n s + \omega_n^2} + \frac{\dot{x}(0)}{s^2 + 2.5\omega_n s + \omega_n^2}$$

$$\mathcal{L}^{-1} \frac{1}{s^2 + 2.5\omega_n s + \omega_n^2} = \frac{1}{\omega_n \sqrt{1-2.5^2}} e^{-2.5\omega_n t} \sin \sqrt{1-2.5^2} \omega_n t \quad \text{see Append. B.}$$

$$\mathcal{L}^{-1} \frac{s}{s^2 + 2.5\omega_n s + \omega_n^2} = \text{derivative of above} = e^{-2.5\omega_n t} \left\{ \cos \sqrt{1-2.5^2} \omega_n t - \frac{2.5\omega_n}{\omega_n \sqrt{1-2.5^2}} \sin \sqrt{1-2.5^2} \omega_n t \right\}$$

$$\therefore x(t) = e^{-2.5\omega_n t} \left\{ \frac{\dot{x}(0) + 2.5\omega_n x(0)}{\omega_n \sqrt{1-2.5^2}} \sin \sqrt{1-2.5^2} \omega_n t + x(0) \cos \sqrt{1-2.5^2} \omega_n t \right\}$$

4-10 $\ddot{y}(t) = 20u(t) - 100t \quad \therefore \ddot{y}(t) = 20\delta(t) - 100$

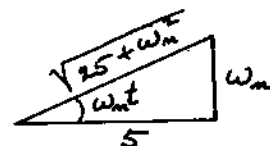
$$\ddot{z} + \omega_n^2 z = -\ddot{y} = 100 - 20\delta(t)$$

$$\bar{z}(s) = \frac{100}{s(s^2 + \omega_n^2)} - \frac{20}{(s^2 + \omega_n^2)}$$

$$z(t) = \frac{100}{\omega_n^2} (1 - \cos \omega_n t) - \frac{20}{\omega_n} \sin \omega_n t$$

$$\dot{z}(t) = \frac{100}{\omega_n} \sin \omega_n t - 20 \cos \omega_n t = 0 \quad \text{for max } z$$

$$\therefore \tan \omega_n t = \frac{\omega_n}{5} \quad (\text{see Fig})$$



$$z_{\max} = \frac{100}{\omega_n} \left[1 - \frac{5}{\sqrt{25 + \omega_n^2}} \right] - \frac{20}{\omega_n} \frac{\omega_n}{\sqrt{25 + \omega_n^2}} \quad \omega_n = 10$$

4-11 $F_0 \sin \omega t = F_0 \sin \frac{\pi}{t_1} t$

$$\ddot{x} + \omega_m^2 x = \frac{F_0}{m} \sin \frac{\pi}{t_1} t \quad \omega_m = \frac{2\pi}{\tau}$$

Gen Sol. $x(t) = A \sin \omega_m t + B \cos \omega_m t + \frac{F_0/m \sin \frac{\pi}{t_1} t}{1 - (\frac{\pi}{t_1} \omega_m)^2}$

For $x(0) = \dot{x}(0) = 0$, $B = 0$ and $A = -\frac{F_0/m (\frac{\pi}{t_1} \omega_m)}{1 - (\frac{\pi}{t_1} \omega_m)^2}$

With $\frac{\pi}{t_1} \omega_m = \frac{\pi}{t_1} \frac{\tau}{2\pi} = \frac{\tau}{2t_1}$

$$x(t) = \frac{F_0}{m} \left\{ \frac{-\frac{\tau}{2t_1} \sin \frac{2\pi t}{\tau}}{1 - (\frac{\tau}{2t_1})^2} + \frac{\sin \frac{\pi t}{t_1}}{1 - (\frac{\tau}{2t_1})^2} \right\}$$

$$= \frac{F_0/m}{(\frac{\tau}{2t_1} - \frac{2t_1}{\tau})} \left\{ \sin \frac{2\pi t}{\tau} - \frac{2t_1}{\tau} \sin \frac{\pi t}{t_1} \right\} \quad t < t_1$$

For $t > t_1$, add same solution with t replaced by $(t - t_1)$

4-12 The triangular force can be represented by

$$F_1 = \frac{2F_0}{t_1} t \quad 0 < t < t_{1/2}$$

$$F_2 = -\frac{4F_0}{t_1} (t - t_{1/2}) + F_1 \quad t_{1/2} < t < t_1$$

$$F_3 = \frac{2F_0}{t_1} (t - t_1) + F_2 \quad t_1 < t$$

Diff. Eq. for $0 < t < t_{1/2}$ is

$$\ddot{x} + \omega_m^2 x = \frac{2F_0}{m t_1} t = \left(\frac{2\omega_m^2 F_0}{k t_1} \right) t = C t$$

$$\bar{x}(s) = \frac{C}{s^2(s^2 + \omega_m^2)} \quad \therefore x_1(t) = \frac{C}{\omega_m^3} (\omega_m t - \sin \omega_m t) = \frac{2F_0}{k} \left(\frac{t}{t_1} - \frac{\tau}{2\pi t_1} \sin \frac{2\pi t}{\tau} \right)$$

For F_2 additional excitation is $-2Ct + Ct_1 = -2C(t - t_{1/2})$

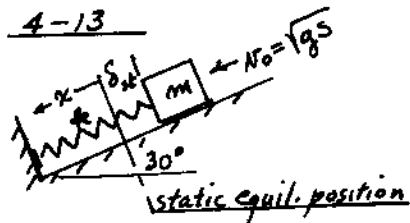
Thus $x_1(t)$ must be supplemented by $-2x_1(t - t_{1/2})$

$$x_2(t) = x_1(t) - 2x_1(t - \frac{1}{2}t_1) \quad \frac{1}{2}t_1 < t < t_1$$

For F_3 the excitation in addition to F_2 is $C(t - t_1)$

$$x_3(t) = x_2(t) + x_1(t - t_1) \quad t_1 < t$$

4-13



$$x = x(0) \cos \omega_n t + \frac{\dot{x}(0)}{\omega_n} \sin \omega_n t$$

$$\dot{x} = -\omega_n x(0) \sin \omega_n t + \dot{x}(0) \cos \omega_n t$$

$$k \delta_{st} = \frac{1}{2} mg$$

When returned to initial position

$$\left. \begin{aligned} -\delta_{st} &= -\delta_{st} \cos \omega_n t + \frac{v_0}{\omega_n} \sin \omega_n t \\ -v_0 &= \omega_n \delta_{st} \sin \omega_n t + v_0 \cos \omega_n t \end{aligned} \right\} \begin{aligned} &\div \text{ by } \cos \omega_n t \text{ and} \\ &\div \text{ the two eqs.} \end{aligned}$$

$$\frac{\delta_{st}}{v_0} = \frac{-\delta_{st} + \frac{v_0}{\omega_n} \tan \omega_n t}{\omega_n \delta_{st} \tan \omega_n t + v_0}$$

$$\tan \omega_n t = \frac{2 \delta_{st} \omega_n v_0}{v_0^2 - \delta_{st}^2 \omega_n^2}$$

$$\tan \omega_n t = \frac{\sqrt{\frac{m}{k} g S}}{S - \frac{g m}{4k}}$$

$$\omega_n^2 = k/m$$

$$\delta_{st} = \frac{g}{\omega_n^2}$$

$$v_0 = \sqrt{g S}$$

4-14

$$\Delta = \frac{38.6}{6.40} = 6.04'' , \quad \text{Max displ.} = 2\Delta = 12.08''$$

$$\omega_n = \sqrt{\frac{6.40 \times 386}{38.6}} = 8.0 \text{ rad/s.} = \frac{2\pi}{T}$$

$$T = \frac{2\pi}{8} = 0.784 \text{ s.}$$

$$t_{\text{max displ}} = \frac{T}{2} = 0.392 \text{ s.}$$

4-15

$$m \ddot{x} = -k(x-y) - f$$

$$\text{Let } z = x - y$$

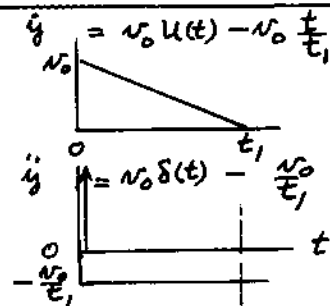
$$\ddot{z} + \omega_n^2 z = -\ddot{y} - \frac{f}{m}$$

Gen Sol. from Eq. (4.2-5) is

$$z = \frac{-1}{\omega_n} \int_0^t \left[\ddot{y}(\xi) + \frac{f}{m} \right] \sin \omega_n(t-\xi) d\xi$$

$$= -\frac{v_0}{\omega_n} \int_0^t \left[\delta(t) - \frac{1}{t_1} + \frac{f}{m v_0} \right] \sin \omega_n(t-\xi) d\xi$$

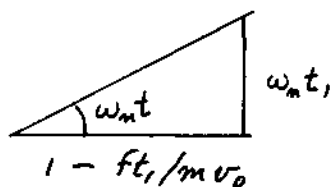
$$= \frac{v_0}{\omega_n} \left\{ \frac{1}{\omega_n t_1} \left(1 - \frac{f t_1}{m v_0} \right) (1 - \cos \omega_n t) - \sin \omega_n t \right\}$$



4-16 Differentiate Z in Prob. 4-15

$$\frac{dz}{dt} = \frac{v_0}{\omega_m} \left\{ \frac{1}{t_1} \left(1 - \frac{ft_1}{mv_0} \right) \sin \omega_m t - \omega_m \cos \omega_m t \right\} = 0$$

$$\therefore \tan \omega_m t = \frac{\omega_m t_1}{1 - \frac{ft_1}{mv_0}}$$



$$\sin \omega_m t = \frac{\omega_m t_1}{\sqrt{(\omega_m t_1)^2 + [1 - ft_1/mv_0]^2}}$$

$$\cos \omega_m t = \frac{(1 - ft_1/mv_0)}{\sqrt{\text{same}}}$$

subst into eq. for Z

$$\begin{aligned} \frac{\omega_m Z_{\max}}{v_0} &= \frac{1}{\omega_m t_1} \left(1 - \frac{ft_1}{mv_0} \right) \left\{ 1 - \frac{(1 - \frac{ft_1}{mv_0})}{\sqrt{(\omega_m t_1)^2 + [1 - ft_1/mv_0]^2}} \right\} - \frac{\omega_m t_1}{\sqrt{(\omega_m t_1)^2 + [1 - ft_1/mv_0]^2}} \\ &= \frac{1}{\omega_m t_1} \left(1 - \frac{ft_1}{mv_0} \right) \left\{ 1 - \frac{\frac{1}{\omega_m t_1} (1 - \frac{ft_1}{mv_0})}{\sqrt{1 + [\frac{1}{\omega_m t_1} (1 - \frac{ft_1}{mv_0})]^2}} \right\} - \frac{1}{\sqrt{1 + [\frac{1}{\omega_m t_1} (1 - \frac{ft_1}{mv_0})]^2}} \end{aligned}$$

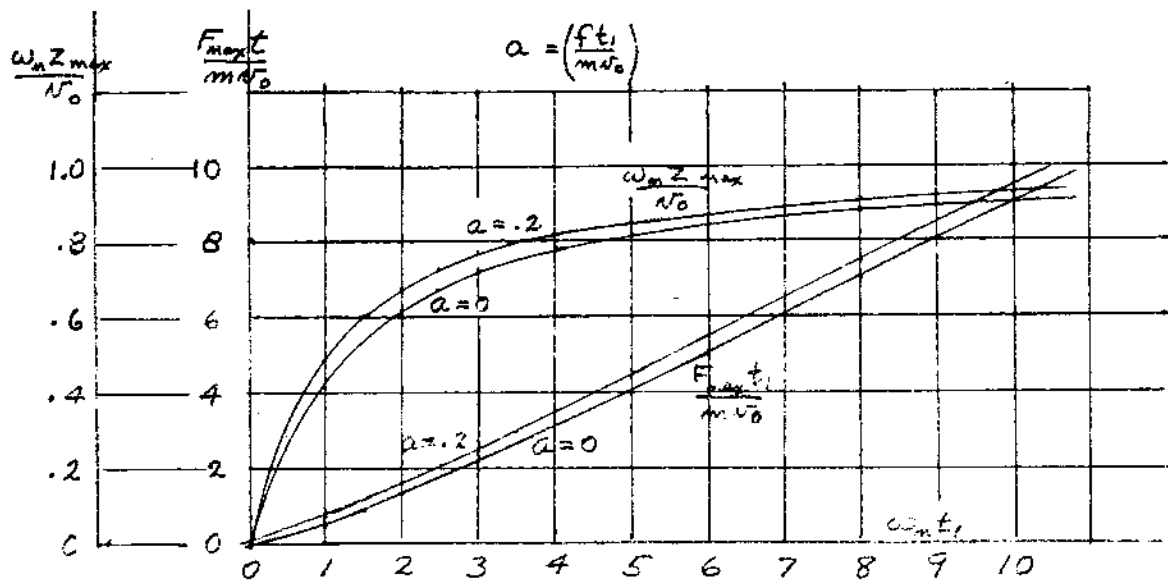
4-17 Let $a = \frac{ft_1}{mv_0}$ and $x = \omega_m t$ $y = \left| \frac{\omega_m Z_{\max}}{v_0} \right|$

$$\left| \frac{\omega_m Z_{\max}}{v_0} \right| = \left| \frac{1}{x} (1-a) \left\{ 1 - \frac{(1-a)}{\sqrt{x^2 + (1-a)^2}} \right\} - \frac{x}{\sqrt{x^2 + (1-a)^2}} \right|$$

$$\left| \frac{F_{\max} t_1}{mv_0} \right| = a + \left| (\omega_m t_1)^2 \left(\frac{\omega_m Z_{\max}}{v_0} \right) \frac{1}{\omega_m t_1} \right| = a + x \left| \left(\frac{\omega_m Z_{\max}}{v_0} \right) \right|$$

x	$y_{a=0}$	$\left \frac{F_{\max} t_1}{mv_0} \right _{a=0}$	$y_{a=.20}$	$\left \frac{F_{\max} t_1}{mv_0} \right _{a=.2}$
1	.4142	.4142	.4806	.6806
2	.6180	1.2360	.6770	1.554
3	.7208	2.1624	.7683	2.504
4	.7808	3.1232	.8198	3.479
5	.8198	4.0990	.8527	4.466
6	.8471	5.0826	.8755	5.453
8	.8828	7.0624	.9050	7.440
10	.9050	9.050	.9232	9.432

4-17 Cont:



4-18 $x = \frac{F_0}{k}(1 - \cos \omega_n t) \quad 0 < t < t_0$

$= \frac{F_0}{k} [\cos \omega_n (t - t_0) - \cos \omega_n t] \quad t > t_0$

For $t < t_0$ $\frac{dx}{dt} = \frac{F_0}{k} \omega_n \sin \omega_n t = 0 \quad \begin{cases} \omega_n t_p = \pi \\ t_p = \frac{1}{2} \end{cases}$

$\left(\frac{xk}{F_0}\right)_{\max} = 2 \quad \text{but } t_p = \frac{1}{2} < t_0$
 $\therefore \frac{t_0}{T} > 0.50$

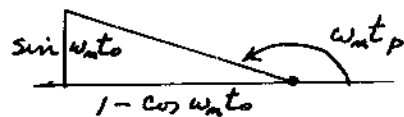
For $t > t_0$ $\frac{dx}{dt} = \frac{F_0 \omega_n}{k} [\sin \omega_n t - \sin \omega_n (t - t_0)] = 0$

$\sin \omega_n t_p - \sin \omega_n t_p \cos \omega_n t_0 + \cos \omega_n t_p \sin \omega_n t_0 = 0$

$\tan \omega_n t_p = \frac{\sin \omega_n t_0}{(1 - \cos \omega_n t_0)}$

$\sin \omega_n t_p = \frac{\sin \omega_n t_0}{\sqrt{2(1 - \cos \omega_n t_0)}}$

$\cos \omega_n t_p = \frac{-(1 - \cos \omega_n t_0)}{\sqrt{\text{same}}}$



Subst into eq. for $t > t_0$

$\left(\frac{kx}{F_0}\right)_{\max} = \sqrt{2(1 - \cos \omega_n t_0)} = 2 \sin \frac{1}{2} \omega_n t_0 = 2 \sin \frac{\pi t_0}{T}$
 $\frac{t_0}{T} < 0.50$

4-19 For small $\frac{t_1}{\tau}$ the sine pulse approaches an impulse

$$\hat{F} = F_0 \int_0^{t_1} \sin \frac{\pi t}{t_1} dt = \frac{2}{\pi} F_0 t_1$$

$$\text{Response} = x = \frac{\hat{F}}{m \omega_n} \sin \omega_n t = \frac{F_0}{k} \frac{4 t_1}{\tau} \sin \frac{\lambda \pi t}{\tau}$$

$$\left(\frac{x k}{F_0} \right)_{\max} = \frac{4 t_1}{\tau} \quad \text{at } t_p = \tau/4 \quad \text{Peak response must occur at time } \gg t_1 \therefore \frac{t_1}{\tau} \ll \frac{1}{4}$$

$$\text{Let } \alpha = \frac{t_1}{\tau}, \quad \xi = \frac{t}{t_1} \quad \text{for } t \geq t_1$$

$$\left(\frac{x k}{F_0} \right) = \frac{1}{\left(\frac{1}{2\alpha} - 2\alpha \right)} \left[\sin 2\pi\alpha\xi + \sin 2\pi\alpha(\xi-1) \right]$$

$$\text{When } \alpha = \frac{1}{2} \quad \text{above eq. is indeterminate} = \frac{0}{0}$$

\therefore differentiate num. & den, w.r.t. α & divide

$$\left(\frac{x k}{F_0} \right)_{\alpha=\frac{1}{2}} = -\frac{\pi}{2} \cos \pi \xi \quad \text{which is max when } \xi=1 = \frac{t}{t_1}$$

$$\therefore t_p = t_1$$

$$\text{and } \left(\frac{x k}{F_0} \right)_{\alpha=\frac{1}{2}, \xi=1} = \frac{\pi}{2} = \underline{\underline{1.57}}$$

$$\underline{4-20} \quad m = \frac{16.1}{386} = 0.0417 \quad \frac{t_1}{\tau} = \frac{0.40}{0.50} = 0.80$$

$$\text{From Fig P4-21} \quad \left(\frac{k x}{F_0} \right)_{\max} = 1.54$$

$$\omega_n = \frac{2\pi}{0.50} = 4\pi \quad k = m \omega_n^2 = 0.0417 (4\pi)^2 = 6.585 \text{ #/in}$$

$$x_{\max} = 1.54 \frac{F_0}{k}$$

$$\hat{F} = 2.0 \text{ lb.} = \frac{1}{2} \times 0.40 \times F_0$$

$$\therefore F_0 = 10$$

$$x_{\max} = 1.54 \times \frac{10}{6.585} = \underline{\underline{2.339''}}$$

4-21 Differentiate 3rd eq. Prob. 4-12

$$\frac{dx}{dt} = \frac{2F_0}{kt_1} \left\{ 2 \cos \frac{2\pi t_1}{T} \left(\frac{t}{t_1} - 0.5 \right) - \cos \frac{2\pi t_1}{T} \left(\frac{t}{t_1} - 1.0 \right) - \cos \frac{2\pi t}{T} \right\} = 0$$

$$\text{or } 2 \cos \frac{2\pi t_1}{T} \left(\frac{t_p}{t_1} - 0.50 \right) - \cos \frac{2\pi t_1}{T} \left(\frac{t_p}{t_1} - 1 \right) - \cos \frac{2\pi t_p}{T} = 0$$

which gives $t_p = t_1$

4-22 With suggested subst for $A \cos \phi$ and $A \sin \phi$

$$x = \frac{A}{k} \{ \sin \omega_n t \cos \phi - \cos \omega_n t \sin \phi \} = \frac{A}{k} \sin(\omega_n t - \phi)$$

Max response occurs when $(\omega_n t - \phi) = \frac{\pi}{2}$

$$x_{\max} = \frac{A}{k} \quad \text{where} \quad A = \omega_n \sqrt{\left[\int_0^t f(\xi) \sin \omega_n \xi d\xi \right]^2 + \left[\int_0^t f(\xi) \cos \omega_n \xi d\xi \right]^2}$$

4-23

$$F = F_0 \frac{t}{t_0} \quad t < t_0$$

$$= 0 \quad t > t_0$$

Eq. 4.2-1 with $h(t) = \frac{1}{m\omega_n} \sin \omega_n t$

$$x(t) = \frac{F_0}{m\omega_n} \int_0^t \frac{\xi}{t_0} \sin \omega_n (t - \xi) d\xi = \frac{F_0}{k} \left(\frac{t}{t_0} - \frac{\sin \omega_n t}{\omega_n t_0} \right) \quad t < t_0$$

For $t > t_0$ integral does not change after t_0

$$x(t) = \frac{F_0}{m\omega_n} \int_0^{t_0} \frac{\xi}{t_0} \sin \omega_n (t - \xi) d\xi = \frac{F_0}{k} \left[\frac{\xi}{t_0} \cos \omega_n (t - \xi) + \frac{\sin \omega_n (t - \xi)}{\omega_n t_0} \right]_0^{t_0}$$

$$= \frac{F_0}{k} \left[\cos \omega_n (t - t_0) + \frac{1}{\omega_n t_0} \{ \sin \omega_n (t - t_0) - \sin \omega_n t \} \right] \quad t > t_0$$

4-24 Given vel. = $v = v_0 (u(t) - t/t_1)$

$$\text{Accel} = a = v_0 [\delta(t) - 1/t_1] = \ddot{y}(t)$$

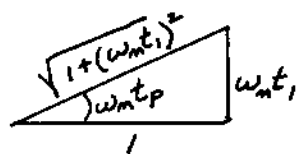
Subst into Eq. 4.2-5

$$z = -\frac{v_0}{\omega_n} \int_0^t \left[\delta(\xi) - \frac{1}{t_1} \right] \sin \omega_n (t - \xi) d\xi \quad 0 < t < t_1$$

$$= \frac{v_0}{\omega_n} \left\{ -\sin \omega_n t + \frac{1}{\omega_n t_1} (1 - \cos \omega_n t) \right\} \quad 0 < t < t_1$$

4-24 Cont:

$$\frac{dz}{dt} = \frac{v_0}{\omega_m} \left\{ -\omega_m \cos \omega_m t + \frac{1}{t_1} \sin \omega_m t \right\} = 0$$



$$\text{or } \tan \omega_m t_p = \omega_m t_1 \quad \text{for } Z_{\max}$$

$$\sin \omega_m t_p = \frac{\omega_m t_1}{\sqrt{1 + (\omega_m t_1)^2}}$$

$$\cos \omega_m t_p = \frac{1}{\sqrt{1 + (\omega_m t_1)^2}}$$

$$\therefore \frac{Z_{\max} \omega_m}{v_0} = \left\{ \frac{1}{\omega_m t_1} - \frac{1}{\omega_m t_1 \sqrt{1 + (\omega_m t_1)^2}} - \frac{\omega_m t_1}{\sqrt{1 + (\omega_m t_1)^2}} \right\}$$

4-25

If $t > t_1$, the solution is

$$\begin{aligned} z &= -\frac{v_0}{\omega_m} \int_0^{t_1} \left[\delta(\xi) - \frac{1}{t_1} \right] \sin \omega_m (t - \xi) d\xi \\ &= \frac{v_0}{\omega_m} \left\{ -\sin \omega_m t + \frac{1}{\omega_m t_1} [\cos \omega_m (t - t_1) - \cos \omega_m t] \right\} \end{aligned}$$

4-26

$$\ddot{z} + \omega_m^2 z = -\ddot{y} = 100 - 20\delta(t)$$

$$\tau = \frac{2\pi}{\omega_m} = 0.628 \text{ s.}$$

$$\text{choose } \Delta t = 0.05 \text{ s}$$

For the impulse response, it is advisable to treat it separately.

$$\text{Its response is } -\frac{20}{\omega_m} \int_0^t \delta(\xi) \sin \omega_m (t - \xi) d\xi = -2 \sin 10t$$

Remaining solution can be computed numerically from D.E

$$\begin{aligned} (a) \quad \ddot{z}_i &= 100 - 100 z_i \\ (b) \quad \ddot{z}_i \Delta t^2 &= z_{i-1} - 2z_i + z_{i+1} \end{aligned} \quad \left. \begin{array}{l} \text{Initial cond.} \\ z_1 = \dot{z}_1 = 0 \end{array} \right\}$$

$$\text{Start with } \ddot{z}_1 = 100 \quad \text{from (a)}$$

$$\text{From Eq(4.5-8)} \quad z_2 = \frac{1}{2} (0.05)^2 100 = 0.1250$$

$$\text{from (a)} \quad \ddot{z}_2 = 100 - 100(0.1250)$$

Find z_3 from Eq. (b) etc.

4-27 This problem is identical to Ex 4.5-2 in the text. It is suggested that the instructor change the triangular impulse duration to 0.80 s, and follow the procedure of Ex. 4.5-2.

4-28 Ex. 4A-2 is for $y = v_0 e^{-t/t_0}$
 Fig P4-28 is for $y = 60 e^{-0.10t}$ applies for $\omega_n t_0$
 or $\omega_n t_0$

$$\therefore v_0 = 60, \quad t_0 = 10$$

For large $\omega_n t_0$ $\left(\frac{2Z}{v_0 t_0}\right)_{\max} \approx \frac{2}{\omega_n t_0}$ a rectangular hyperbola

at $\omega_n t_0 = 100$ Fig 4-28 gives $\left(\frac{2Z}{v_0 t_0}\right)_{\max} = 0.02$

For small $\omega_n t_0$ $\left(\frac{2Z}{v_0 t_0}\right)_{\max} \approx 1.0$

4-29 $m\ddot{x} + c\dot{x} + kx = F_0 \sin \omega_n t$

With zero initial conditions, response may be evaluated from (1)
 (2) or (3) below

$$(1) \quad \bar{x}(s) = \frac{\omega_n F_0 / m}{(s^2 + \omega_n^2)(s^2 + 2\zeta\omega_n s + \omega_n^2)}$$

$$(2) \quad x(t) = X_1 e^{-\zeta\omega_n t} \sin(\sqrt{1-\zeta^2}\omega_n t + \phi_1) - \frac{F_0 \cos \omega_n t}{c\omega_n} \quad \text{Eq. (3.1-11)}$$

$$(3) \quad x(t) = F_0 \int_0^t \sin \omega_n(t-\tau) \cdot \frac{e^{-\zeta\omega_n \tau}}{m\omega_n \sqrt{1-\zeta^2}} \sin \sqrt{1-\zeta^2}\omega_n \tau \quad \text{Eq. (4.2-2)}$$

Result is:

$$x(t) = \frac{F_0}{c\omega_n} \left\{ \frac{e^{-\zeta\omega_n t}}{\sqrt{1-\zeta^2}} \sin(\sqrt{1-\zeta^2}\omega_n t + \sin^{-1}\sqrt{1-\zeta^2}) - \cos \omega_n t \right\}$$

4-30 For small damping $\delta \approx 2\pi\zeta$

At time $t = \frac{1}{f_n \delta}$, $\zeta\omega_n t = \frac{2\pi\zeta}{\delta} \approx 1.0$ Subst. into Eq. above

$$\begin{aligned} x &= \frac{F_0}{c\omega_n} \left\{ e^{-1} \sin(\omega_n t + 90^\circ) - \cos \omega_n t \right\} \\ &= \frac{F_0}{c\omega_n} \left\{ e^{-1} - 1 \right\} \cos \omega_n t = (1 - e^{-1}) \left(\frac{-F}{c\omega_n} \cos \omega_n t \right) \end{aligned}$$

where steady state sol. $= \left(\frac{-F}{c\omega_n} \cos \omega_n t \right)$

4-31 Under harmonic force of freq. ω_m , the steady state oscillation is

$$x(t) = \frac{F_0}{c\omega_m} \cos \omega_m t \quad x(0) = \frac{F_0}{c\omega_m}, \quad \dot{x}(0) = 0$$

Transient sol. is

$$x(t) = X_1 e^{-\zeta \omega_m t} \sin(\sqrt{1-\zeta^2} \omega_m t + \phi_1) \approx X_1 e^{-\zeta \omega_m t} \sin(\omega_m t + \phi_1) \quad \text{for } \zeta \text{ small}$$

$$x(0) = X_1 \sin \phi_1 = \frac{F_0}{c\omega_m}$$

$$\dot{x}(0) = X_1 [\omega_m \cos \phi_1 - \zeta \omega_m \sin \phi_1] \approx X_1 \omega_m \cos \phi_1 = 0$$

$$\therefore \phi_1 = 90^\circ \quad \text{and} \quad X_1 = \frac{F_0}{c\omega_m}$$

Then trans. sol. with above initial cond. is

$$x(t) = \frac{F_0}{c\omega_m} e^{-\zeta \omega_m t} \cos \omega_m t$$

$$\text{At } \zeta \omega_m t = \frac{2\pi\zeta}{\delta} \approx 1.0 \quad x(t) = e^{-1} \frac{F_0}{c\omega_m} \cos \omega_m t$$

see Prob 4-30

4-32

$$\text{DE} \quad \ddot{x} = 0.25 F(t) - 500 x$$

$$\text{Let } h = H = 0.02$$

$$t = T(I) = H * (I-1) \quad I=1, 2, \dots$$

$$x = X(I) \quad X(1) = 0$$

$$\ddot{x} = DX2(I)$$

Let I go from 1 to 25 = N

Computer program follows

4-32 Cont.

DIMENSION T(28), X(28), DX2(28), F(28)

N = 25

H = .02

T(1) = 0

X(1) = 0

DO I = 1, 25

T(I) = H * (I-1)

IF (I.GT.1) GO TO 2

F(I) = 100

DX2(I) = .25 * F(I) - 500 * X(I)

X(I+1) = .50 * DX2(I) * H ** 2

GO TO 3

2 IF (I.LT.6) F(I) = 100

IF (I.GE.6) F(I) = 100 - 1000 * (T(I) - .10)

IF (I.GT.10) F(I) = 0

DX2(I) = .25 * F(I) - 500 * X(I)

X(I+1) = 2 * X(I) - X(I-1) + DX2(I) * H ** 2

IF (I = N+1) GO TO 4

3 CONTINUE

4 WRITE (6, 5)

5 FORMAT (4 I H 1, TIME, FORCE, DISPL)

WRITE (6, 6) (I(I), F(I), X(I))

6 FORMAT (3X, F6.3, 3X, F6.3, 3X, F6.4)

STOP

END

4-33

We will first discuss the case where the initial conditions are not zero. The discussion on p 108 then applies.

The acceleration from the D.E. is

$$\ddot{x}_i = \frac{F_i}{m} - \omega_m^2 x_i - 2\zeta \omega_m \dot{x}_i = f(x_i, \dot{x}_i, t_i)$$

The two equations given from the Taylor series are

$$(a) \quad x_2 = x_1 + \dot{x}_1 h + \frac{h^2}{2} \left(\frac{F_1}{m} - \omega_m^2 x_1 - 2\zeta \omega_m \dot{x}_1 \right)$$

$$(b) \quad \dot{x}_1 = \dot{x}_2 - \ddot{x}_2 h + \frac{h^2}{2} \left(\frac{F_2}{m} - \omega_m^2 x_2 - 2\zeta \omega_m \dot{x}_2 \right)$$

Since the initial conditions x_1 , \dot{x}_1 , and F_1 are assumed to be given, Eq. (a) gives x_2 .

Eq. (b) is next solved for \dot{x}_2 which is

$$(c) \quad \dot{x}_2 = \frac{x_2 - x_1 + \frac{h^2}{2} \left(\frac{F_2}{m} - \omega_m^2 x_2 \right)}{(h + h^2 \zeta \omega_m)}$$

Thus x_2 and \dot{x}_2 for the first interval can be calculated from Eqs. (a) and (c).

Calculations for x_3 , \dot{x}_3 , x_4 , \dot{x}_4 etc are now made with Eq. (4.5-7') and Eq. (c) generalized to index i as follows.

$$(d) \quad x_{i+1} = 2x_i - x_{i-1} + h^2 \left(\frac{F_i}{m} - \omega_m^2 x_i - 2\zeta \omega_m \dot{x}_i \right)$$

$$(e) \quad \dot{x}_{i+1} = \frac{x_{i+1} - x_i + \frac{h^2}{2} \left(\frac{F_{i+1}}{m} - \omega_m^2 x_{i+1} \right)}{(h + h^2 \zeta \omega_m)}$$

Flow diagram follows

4-33 Cont.:

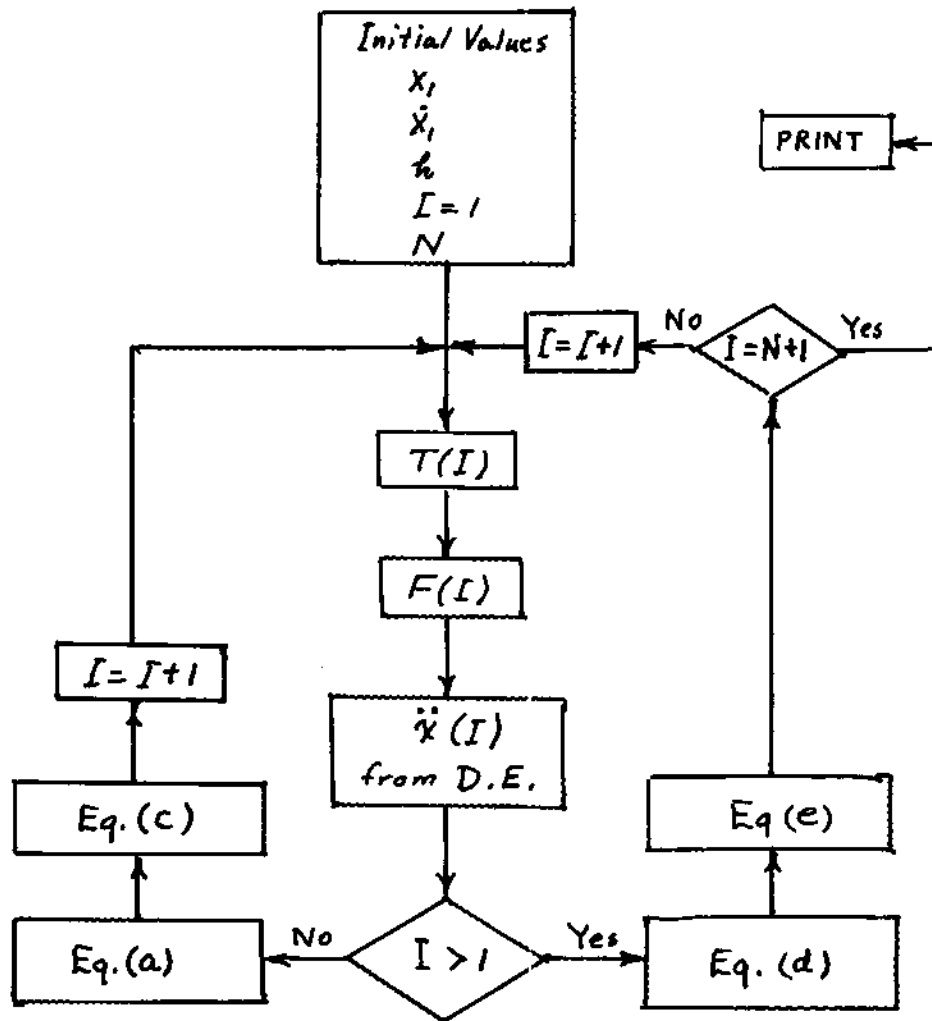


Diagram is similar to FIG. 4.5-1 with added boxes for Eq.(c) and Eq.(e) for the calculation of velocities.

4-33 Cont:

When initial values are zero and force = 0
Eq. (a) gives $x_2 = 0$ and hence calculations
cannot be started. We must then use Eqs 4.5-9
and 4.5-10 and substitute \ddot{x}_2 from D.E. The
two equations so obtained must be solved simultaneously
for x_2 and \dot{x}_2 . With these values calculations
may proceed with Eqs. (d) and (e)

$$x_2 = \frac{h^2}{6} \left(\frac{F_2}{m} - \omega_n^2 x_2 - 25\omega_n \dot{x}_2 \right) \quad (4.5-10)$$

$$\dot{x}_2 = \frac{h}{2} \left(\frac{F_2}{m} - \omega_n^2 x_2 - 25\omega_n \dot{x}_2 \right) \quad (4.5-9)$$

Rearrange to

$$\left(1 + \frac{h^2}{6}\omega_n^2\right)x_2 + \left(\frac{h^2}{3}5\omega_n\right)\dot{x}_2 = \frac{h^2}{6}\left(\frac{F_2}{m}\right)$$

$$\left(\frac{h}{2}\omega_n^2\right)x_2 + (1 + h5\omega_n)\dot{x}_2 = \frac{h}{2}\left(\frac{F_2}{m}\right)$$

Solution:

$$x_2 = \frac{\frac{h^2}{6}\left(\frac{F_2}{m}\right)}{1 + h\omega_n\left(\frac{h}{6} + 5\right)} \quad (f)$$

$$\dot{x}_2 = \frac{\frac{h}{2}\left(\frac{F_2}{m}\right)}{1 + h\omega_n\left(\frac{h}{6} + 5\right)} \quad (g)$$

Previous flow diagram can now be used
with Eq.(a) replaced by Eq.(f) and Eq.(c)
replaced by Eq.(g). Right loop is unaltered.
Calculation of F_2 is necessary prior to Eq.(a).

4-34

DE. for base excitation is

$$m\ddot{x} + c(\dot{x} - \dot{y}) + k(x - y) = 0$$

Let $z = x - y$, then

$$\ddot{z} = -\ddot{y} - \omega_n^2 z - 2\zeta\omega_n \dot{z}$$

Thus the problem is same as that for force excitation with z replacing x and $-\ddot{y}(t)$ replacing $(\frac{F}{m})$.

4-35

Refer to Prob. 4-33 and the given flow diagram. The exciting force $\frac{F}{m}$ is now replaced by $-\ddot{y}$

$$y = y_0 \sin \omega t$$

$$\frac{F}{m} = -\ddot{y} = \omega^2 y_0 \sin \omega t$$

The sine pulse terminates at $\omega t_p = \pi$. The time increment h should be chosen smaller than $\frac{t_p}{10}$ or $\frac{1}{10} \left(\frac{2\pi}{\omega_n} \right)$, whichever is the smallest.

A program similar to Prob. 4-32 can be written with the IF statements modified.

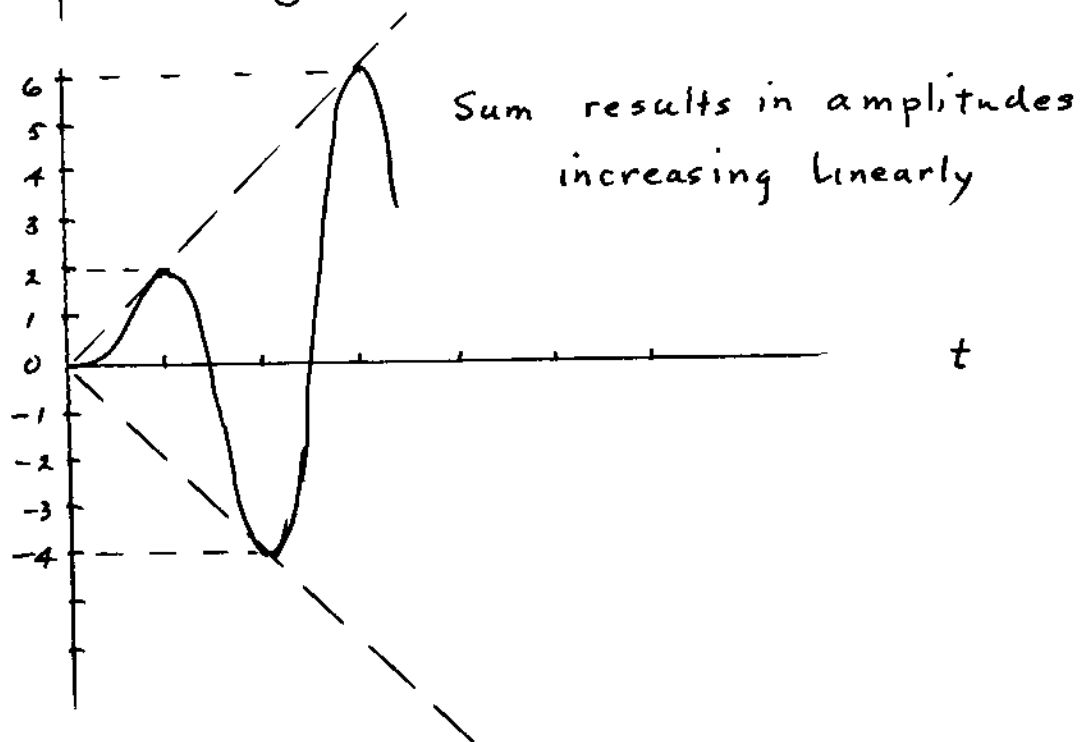
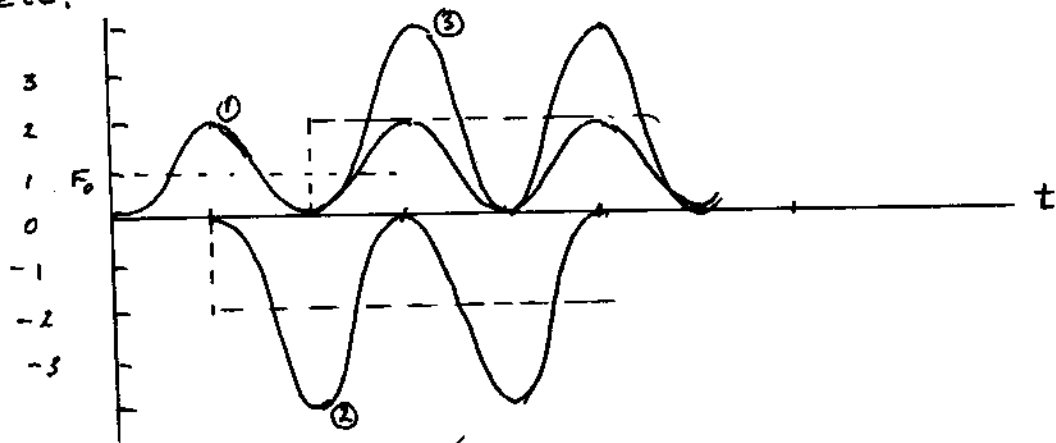
4-36 use superposition of solution to each step function

1st step of F_0 results in $x(t) = \frac{F_0}{k} (1 - \cos \omega_n t)$

2nd step of $-2F_0$ at $t = \frac{\pi}{\omega}$ adds $-\frac{2F_0}{k} [1 - \cos \omega_n (t - \frac{\pi}{\omega})]$

3rd step of $+2F_0$ at $t = \frac{2\pi}{\omega}$ adds $+\frac{2F_0}{k} [1 - \cos \omega_n (t - \frac{2\pi}{\omega})]$

etc.



4-37

$$x_{i+1} = x_i + \dot{x}_i h + \ddot{x}_i \frac{h^2}{2} + \ddot{\ddot{x}}_i \frac{h^3}{6} + \ddot{\ddot{\ddot{x}}}_i \frac{h^4}{24} + \dots$$

$$x_{i-1} = x_i - \dot{x}_i h + \ddot{x}_i \frac{h^2}{2} - \ddot{\ddot{x}}_i \frac{h^3}{6} + \ddot{\ddot{\ddot{x}}}_i \frac{h^4}{24} - \dots$$

$$x_{i+1} + x_{i-1} = 2x_i + \ddot{x}_i h^2 + \ddot{\ddot{\ddot{x}}}_i \frac{h^4}{12}$$

$$\ddot{\ddot{\ddot{x}}}_i = \frac{x_{i-1} - 2x_i + x_{i+1}}{h^2} - \ddot{\ddot{\ddot{x}}}_i \frac{h^2}{12}$$

$$\text{error} = -\frac{\ddot{\ddot{\ddot{x}}}_i}{12} h^2 = O(h^2)$$

4-38

t	$x = t^3$
.8	.5120
.9	.7290
1.0	1.0000
1.1	1.3310
1.2	1.7280

Exact value of \dot{x} is

$$\dot{x} = 3t$$

$$\therefore \dot{x}(1) = 3.0000$$

Calculation of \dot{x} by finite difference

using $\dot{x}_i = \frac{1}{2h} (x_{i+1} - x_{i-1})$

$$h=0.10 \quad \dot{x}_1 = \frac{1}{2 \times .10} (1.3310 - .7290) = 3.010$$

$$\therefore \text{Error} = .01 = .1^2 = h^2 = O(h^2)$$

$$h=0.20 \quad \dot{x}_1 = \frac{1}{2 \times .20} (1.7280 - .5120) = 3.04$$

$$\therefore \text{Error} = .04 = .2^2 = h^2 = O(h^2)$$

4-39

Refer to Prob. 4-38 but use

$$\dot{x}_i = \frac{1}{h} (x_i - x_{i-1})$$

$$h = .10 \quad \dot{x}_1 = \frac{1}{.10} (1.0 - .7290) = 2.71$$

$$\therefore \text{Error} = .290 = 2.9 h = O(h)$$

$$h = .20 \quad \dot{x}_1 = \frac{1}{.20} (1.0 - .5120) = 2.44$$

$$\therefore \text{Error} = .560 = 2.8 h = O(h)$$

4-40

$$\text{D.E.} \quad 4\ddot{x} + 2000x = F(t)$$

$$\therefore m = 4, \quad k = 2000$$



$$\omega_n = \sqrt{\frac{k}{m}} = 22.36$$

1st sol. due to step function (see Fig 4.5-4)

$$x_1 = \frac{F}{k} (1 - \cos \omega_n t) = \frac{100}{2000} (1 - \cos 22.36 t)$$

$$0 \leq t \leq 0.10$$

2nd sol. due to ramp function of

$$1000(t - .10) \quad \text{at } t \geq .10$$

$$\text{DE} \quad \ddot{x} + 500x = 250 t' \quad \text{where } t' = t - .10$$

$$\text{Lapl. Trans} \quad x(s) = \frac{250}{s^2(s^2 + 22.36^2)}$$

$$\begin{aligned} x_2 &= \frac{250}{22.36^2} [22.36 t' - \sin 22.36 t'] \\ &= \frac{1}{2} t' - .02236 \sin 22.36 t' \end{aligned}$$

4-40

∴ add to first sol x_1 , the second sol. which

is
$$x_2 = \frac{1}{2}(t-.10) - .02236 \sin 22.36(t-.10)$$

$$\text{at } t \geq .10$$

Similarly the third sol. is same as x_2
with $(t-.20)$

4-41

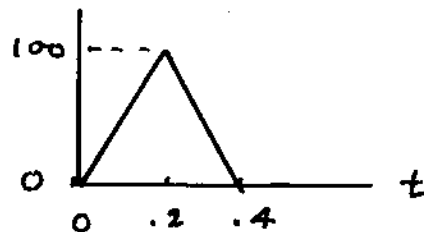
DE to be solved is

$$\ddot{x} + 16\pi^2 x = 2F(t)$$

Let $y = \dot{x}$, then

$$\dot{y} = f(x, t) = 2F(t) - 16\pi^2 x$$

where $F(t) =$



Follow Calc. procedure of Example 4.6-1

$$\omega_n = 4\pi = 12.56 = \frac{2\pi}{\gamma} \quad \gamma = .500$$

∴ suggest $h = .02$

4-41 Cont.Runge-Kutta Program

```

PROBLEM 4-41 THOMSON
DIMENSION T(50), T1(50), T2(50), T3(50), T4(50), X(50), X1(50), X2(50),
1X3(50), X4(50), Y(50), Y1(50), Y2(50), Y3(50), Y4(50), F(50), F1(50),
1F2(50), F3(50), F4(50)
N=49
DH=0.02
X(1)=0.0
Y(1)=0.0
T(1)=0.0
PRINT5
5 FORMAT(20X,'J',5X,'TIME',9X,'DISPL',5X,'ACCELERATION',11X,'F(J)',
110X,'FORCE')
DO 10 J=1,N
F(J)=FXY(T(J),X(J),Y(J),DF)
PRINT8,J,T(J),X(J),Y(J),F(J),DF
8 FORMAT(18X,I3,2X,F7.3,2X,E12.3,5X,E12.3,3X,E12.3,7X,F8.3)
T1(J)=T(J)
X1(J)=X(J)
Y1(J)=Y(J)
F1(J)=FXY(T1(J),X1(J),Y1(J),DF)
T2(J)=T(J)+DH/2.
X2(J)=X(J)+Y1(J)*DH/2.
Y2(J)=Y(J)+F1(J)*DH/2.
F2(J)=FXY(T2(J),X2(J),Y2(J),DF)
T3(J)=T(J)+DH/2.
X3(J)=X(J)+Y2(J)*DH/2.
Y3(J)=Y(J)+F2(J)*DH/2.
F3(J)=FXY(T3(J),X3(J),Y3(J),DF)
T4(J)=T(J)+DH
X4(J)=X(J)+Y3(J)*DH
Y4(J)=Y(J)+F3(J)*DH
F4(J)=FXY(T4(J),X4(J),Y4(J),DF)
X(J+1)=X(J)+DH/6.*(Y1(J)+2.*Y2(J)+2.*Y3(J)+Y4(J))
Y(J+1)=Y(J)+DH/6.*(F1(J)+2.*F2(J)+2.*F3(J)+F4(J))
T(J+1)=T(J)+DH
10 CONTINUE
STOP
END

FUNCTION FXY(T,X,Y,DF)
IF(T.GT.0.4) GO TO 50
IF(T.GT.0.2) GO TO 49
DF=500.*T
GO TO 51
49 DF=200.-500.*T
GO TO 51
50 DF=0.0
51 FXY=2.*DF-16.*3.1415**2*X
RETURN
END

```

4-4/ Cont.

TIME	DISPL	ACCELERATION	F (J)	J	FORCE
0.000	0.000E 00	0.000E 00	0.000E 00	1	0.000
0.020	0.133E-02	0.199E 00	0.198E 02	2	10.000
0.040	0.105E-01	0.783E 00	0.383E 02	3	20.000
0.060	0.350E-01	0.172E 01	0.545E 02	4	30.000
0.080	0.811E-01	0.294E 01	0.672E 02	5	40.000
0.100	0.154E 00	0.438E 01	0.757E 02	6	50.000
0.120	0.257E 00	0.593E 01	0.794E 02	7	60.000
0.140	0.392E 00	0.752E 01	0.782E 02	8	70.000
0.160	0.557E 00	0.903E 01	0.720E 02	9	80.000
0.180	0.752E 00	0.104E 02	0.613E 02	10	90.000
0.200	0.970E 00	0.115E 02	0.468E 02	11	100.000
0.220	0.120E 01	0.118E 02	-0.103E 02	12	90.000
0.240	0.144E 01	0.110E 02	-0.667E 02	13	80.000
0.260	0.164E 01	0.918E 01	-0.119E 03	14	70.000
0.280	0.180E 01	0.634E 01	-0.164E 03	15	60.000
0.300	0.189E 01	0.271E 01	-0.198E 03	16	50.000
0.320	0.190E 01	-0.150E 01	-0.220E 03	17	40.000
0.340	0.183E 01	-0.601E 01	-0.228E 03	18	30.000
0.360	0.166E 01	-0.105E 02	-0.222E 03	19	20.000
0.380	0.141E 01	-0.148E 02	-0.202E 03	20	10.000
0.400	0.107E 01	-0.185E 02	-0.169E 03	21	0.000
0.420	0.671E 00	-0.213E 02	-0.106E 03	22	0.000
0.440	0.229E 00	-0.227E 02	-0.361E 02	23	0.000
0.460	-0.228E 00	-0.227E 02	0.360E 02	24	0.000
0.480	-0.671E 00	-0.213E 02	0.106E 03	25	0.000
0.500	-0.107E 01	-0.185E 02	0.169E 03	26	0.000
0.520	-0.140E 01	-0.146E 02	0.222E 03	27	0.000
0.540	-0.165E 01	-0.976E 01	0.260E 03	28	0.000
0.560	-0.179E 01	-0.430E 01	0.283E 03	29	0.000
0.580	-0.182E 01	0.143E 01	0.287E 03	30	0.000
0.600	-0.173E 01	0.707E 01	0.274E 03	31	0.000
0.620	-0.154E 01	0.123E 02	0.243E 03	32	0.000
0.640	-0.125E 01	0.167E 02	0.197E 03	33	0.000
0.660	-0.879E 00	0.201E 02	0.139E 03	34	0.000
0.680	-0.454E 00	0.222E 02	0.717E 02	35	0.000
0.700	-0.712E-03	0.229E 02	0.112E 00	36	0.000
0.720	0.453E 00	0.222E 02	-0.715E 02	37	0.000
0.740	0.878E 00	0.201E 02	-0.139E 03	38	0.000
0.760	0.125E 01	0.167E 02	-0.197E 03	39	0.000
0.780	0.154E 01	0.123E 02	-0.243E 03	40	0.000
0.800	0.173E 01	0.709E 01	-0.274E 03	41	0.000
0.820	0.192E 01	0.145E 01	-0.287E 03	42	0.000
0.840	0.179E 01	-0.428E 01	-0.283E 03	43	0.000
0.860	0.165E 01	-0.974E 01	-0.261E 03	44	0.000
0.880	0.141E 01	-0.146E 02	-0.222E 03	45	0.000
0.900	0.107E 01	-0.185E 02	-0.169E 03	46	0.000
0.920	0.672E 00	-0.213E 02	-0.106E 03	47	0.000
0.940	0.230E 00	-0.227E 02	-0.362E 02	48	0.000
0.960	-0.227E 00	-0.227E 02	0.359E 02	49	0.000

The stiffness of the crane boom is represented by k_c , measured from the extended straight line.

$$\frac{W}{g} \ddot{y} = k_c (x - y) - W$$

$$\ddot{y} + \left(\frac{k_c g}{W}\right) y = \left(\frac{k_c g}{W}\right) x - g, \quad \omega^2 = \left(\frac{k_c g}{W}\right)$$

$$\ddot{y} + \omega^2 y = \omega^2 Vt - g, \quad x = Vt$$

By L.T.

$$s^2 \bar{y}(s) - s y(0) - \dot{y}(0) + \omega^2 \bar{y}(s) = \frac{\omega^2 V}{s^2} - \frac{g}{s}$$

$$\bar{y}(s) = \frac{s y(0)}{s^2 + \omega^2} + \frac{\dot{y}(0)}{s^2 + \omega^2} + \frac{\omega^2 V}{s^2(s^2 + \omega^2)} - \frac{g}{s(s^2 + \omega^2)}$$

Inverse L.T.

$$y(t) = y(0) \cos \omega t + \frac{\dot{y}(0)}{\omega} \sin \omega t$$

$$+ \frac{V}{\omega} (\omega t - \sin \omega t) - \frac{g}{\omega^2} (1 - \cos \omega t)$$

5-1

$$m\ddot{x}_1 = -kx_1 + k(x_2 - x_1)$$

$$m\ddot{x}_2 = -k(x_2 - x_1)$$

origin of x_1, x_2
= equilib position

$$\left. \begin{aligned} (2 - \frac{m\omega^2}{k})x_1 &= x_2 \\ x_1 &= (1 - \frac{m\omega^2}{k})x_2 \end{aligned} \right\} \text{let } \lambda = \frac{m\omega^2}{k}$$

characteristic eq. $\lambda^2 - 3\lambda + 1 = 0$

$$\lambda = \begin{cases} 0.382 & = \frac{m\omega_1^2}{k} \\ 2.618 & = \frac{m\omega_2^2}{k} \end{cases} \quad \begin{aligned} (X_1/X_2)_1 &= 1 - \lambda_1 = 0.614 \\ (X_1/X_2)_2 &= 1 - \lambda_2 = -1.618 \end{aligned}$$

5-2 & 3

$$\begin{bmatrix} m & 0 \\ 0 & m \end{bmatrix} \begin{Bmatrix} \ddot{x}_1 \\ \ddot{x}_2 \end{Bmatrix} + \begin{bmatrix} (k+nk) & -nk \\ -nk & (k+nk) \end{bmatrix} \begin{Bmatrix} x_1 \\ x_2 \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \end{Bmatrix}$$

Let $\lambda = \frac{m\omega^2}{k}$

$$\begin{vmatrix} (1+n-\lambda) & -n \\ -n & (1+n-\lambda) \end{vmatrix} = 0$$

$$\lambda^2 - 2(1+n)\lambda + (1+2n) = 0$$

$$\lambda = (1+n) \pm n, \quad X_1/X_2 = \frac{1+n-\lambda}{n}$$

For $n=1$, $\lambda_1 = 1 = \frac{m\omega_1^2}{k}$ $(X_1/X_2)_1 = 1$

$\lambda_2 = 3 = \frac{m\omega_2^2}{k}$ $(X_1/X_2)_2 = -1$

5-4

$$m \begin{bmatrix} 3 & 0 \\ 0 & 1 \end{bmatrix} \begin{Bmatrix} \ddot{x}_1 \\ \ddot{x}_2 \end{Bmatrix} + k \begin{bmatrix} 2 & -1 \\ -1 & 4 \end{bmatrix} \begin{Bmatrix} x_1 \\ x_2 \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \end{Bmatrix}$$

$\lambda = \frac{m\omega^2}{k}$

$$\begin{vmatrix} (2-3\lambda) & -1 \\ -1 & (4-\lambda) \end{vmatrix} = 0$$

$$\lambda^2 - \frac{14}{3}\lambda + \frac{7}{3} = 0$$

$$\lambda = \frac{14}{6} \pm \sqrt{\left(\frac{14}{6}\right)^2 - \frac{7}{3}} = \begin{cases} 0.570 = \frac{m\omega_1^2}{k} \\ 4.096 = \frac{m\omega_2^2}{k} \end{cases}$$

$$\frac{X_1}{X_2} = (4-\lambda) \quad \left(\frac{X_1}{X_2}\right)_1 = 3.43 \quad \left(\frac{X_1}{X_2}\right)_2 = -0.096$$

5-5

$$J_2 \begin{bmatrix} 2 & 0 \\ 0 & 1 \end{bmatrix} \begin{Bmatrix} \ddot{\theta}_1 \\ \ddot{\theta}_2 \end{Bmatrix} + K \begin{bmatrix} 2 & -1 \\ -1 & 1 \end{bmatrix} \begin{Bmatrix} \theta_1 \\ \theta_2 \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \end{Bmatrix} \quad \lambda = \frac{\omega^2 J_2}{K}$$

$$\begin{vmatrix} (2-2\lambda) & -1 \\ -1 & (1-\lambda) \end{vmatrix} = 0$$

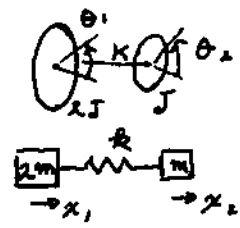
$$2(1-\lambda)^2 - 1 = 0$$

$$(1-\lambda) = \pm 1/\sqrt{2}$$

$$\lambda = 1 \mp 1/\sqrt{2}$$

$$\lambda_{1,2} = \begin{cases} .293 = \frac{J_2 \omega_1^2}{K} \\ 1.707 = \frac{J_2 \omega_2^2}{K} \end{cases} \quad \begin{aligned} \left(\frac{\theta_1}{\theta_2}\right)_1 &= 0.707 \\ \left(\frac{\theta_1}{\theta_2}\right)_2 &= -0.707 \end{aligned}$$

5-6



for $k_1 = 0$

$$\begin{vmatrix} (1-\lambda) & -1 \\ -1 & (1-\lambda) \end{vmatrix} = 0$$

$$\lambda(2\lambda-3) = 0$$

$$\therefore \lambda = 0, \quad \lambda = 1.5$$

$$\frac{\theta_1}{\theta_2} = -\frac{1}{2}$$

↑ free rotation

In general

$$J_1 \ddot{\theta}_1 - k_2 (\theta_2 - \theta_1) = 0$$

$$J_2 \ddot{\theta}_2 + k_2 (\theta_2 - \theta_1) = 0$$

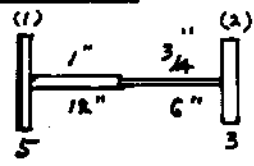
$$(\ddot{\theta}_2 - \ddot{\theta}_1) + \left(\frac{k_2}{J_2} + \frac{k_2}{J_1} \right) (\theta_2 - \theta_1) = 0$$

$$\text{Let } \phi = \pm (\theta_2 - \theta_1)$$

$$\ddot{\phi} + k_2 \left(\frac{J_1 + J_2}{J_1 J_2} \right) \phi = 0$$

$$\omega_n = \sqrt{k_2 \left(\frac{J_1 + J_2}{J_1 J_2} \right)}$$

5-7



$$K_1 = \frac{GJ_P}{L} = \frac{(11.5 \times 10^6)}{12} \frac{\pi 1^4}{32} = 0.0941 \times 10^6$$

$$K_2 = \frac{(11.5 \times 10^6)}{6} \frac{\pi (\frac{3}{4})^4}{32} = 0.0595 \times 10^6$$

$$K_{eff} = \frac{K_1 K_2}{K_1 + K_2} = 0.0365 \times 10^6$$

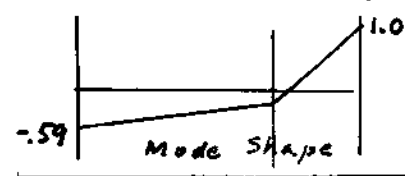
$$\omega_n = \sqrt{K_{eff} \left(\frac{J_1 + J_2}{J_1 J_2} \right)} = \sqrt{\frac{8}{15} \times 0.0365 \times 10^6} = 139.4 \text{ rad/s.}$$

characteristic eq.

$$\begin{vmatrix} (1 - \frac{\omega^2 J_1}{K_{eff}}) & -1 \\ -1 & (1 - \frac{\omega^2 J_2}{K_{eff}}) \end{vmatrix} = 0 \quad \left(\frac{\theta_1}{\theta_2} \right) = \left(1 - \frac{\omega^2 J_2}{K_{eff}} \right) = 1 - \frac{139.4^2 \cdot 3}{0.0365 \times 10^6}$$

$$= 1 - 1.5979 = -0.5979$$

Twist in each shaft \propto to $\frac{1}{K_i}$

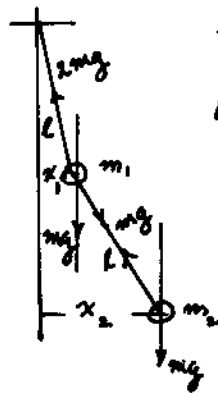


5-8

Total stiffness between cars = 16,000 #/in. System has node at middle of spring \therefore oscillates as two single deg. freed. system with each spring of $2k = 32,000$ #/in and mass m

$$\omega_n = \sqrt{\frac{2k}{m}} = \sqrt{k \left(\frac{m_1 + m_2}{m_1 m_2} \right)} = \sqrt{\frac{2 \times 16,000 \times 386}{50,000}} = 15.72 \text{ rad/s.}$$

5-9



For small amplitudes angles are $\frac{x_1}{l}$ and $\frac{x_2 - x_1}{l}$.
Tensions are approx $2mg$ and mg .

Use $\sum F_x$

$$m\ddot{x}_1 = -2mg\left(\frac{x_1}{l}\right) + mg\left(\frac{x_2 - x_1}{l}\right)$$

$$m\ddot{x}_2 = -mg\left(\frac{x_2 - x_1}{l}\right)$$

Let $\lambda = \frac{\omega^2 l}{g}$

$$(3 - \lambda)X_1 - X_2 = 0$$

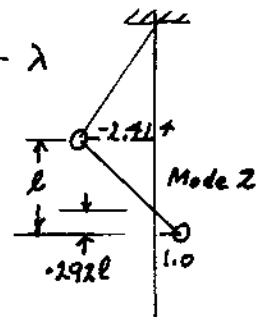
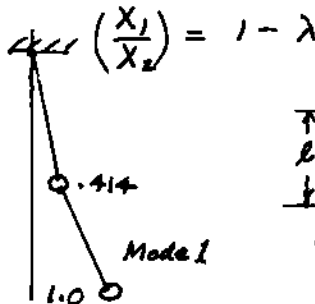
$$-X_1 + (1 - \lambda)X_2 = 0$$

$$\lambda^2 - 4\lambda + 2 = 0, \quad \lambda = 2 \pm \sqrt{2} = \begin{cases} 0.586 \\ 3.414 \end{cases}$$

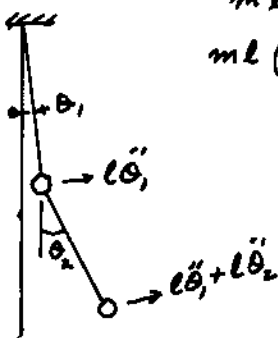
$$\omega_n = \sqrt{(2 \pm \sqrt{2}) \frac{g}{l}} = \begin{cases} 0.764 \sqrt{\frac{g}{l}} \\ 1.850 \sqrt{\frac{g}{l}} \end{cases}$$

$$\left(\frac{X_1}{X_2}\right)_1 = 1 - \lambda_1 = 0.414$$

$$\left(\frac{X_1}{X_2}\right)_2 = -2.414$$



5-10



$$ml\ddot{\theta}_1 = -2mg\theta_1 + mg\theta_2$$

$$ml(\ddot{\theta}_1 + \ddot{\theta}_2) = -mg\theta_2$$

$\lambda = \frac{\omega^2 l}{g}$

$$(2 - \lambda)\theta_1 - \theta_2 = 0$$

$$-\lambda\theta_1 + (1 - \lambda)\theta_2 = 0$$

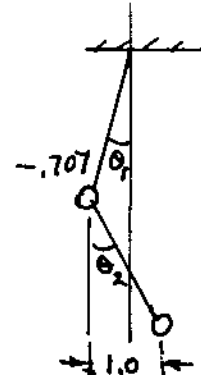
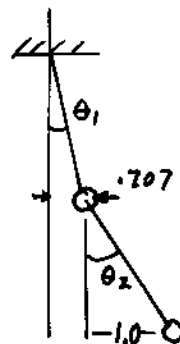
$$\lambda^2 - 4\lambda + 2 = 0$$

Same as Prob 5-9

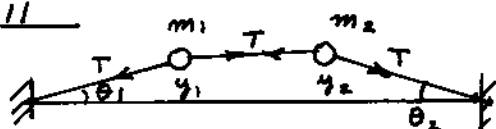
$$\frac{\theta_1}{\theta_2} = \frac{1 - \lambda}{\lambda}$$

$$\begin{Bmatrix} \theta_1 \\ \theta_2 \end{Bmatrix}_1 = \begin{Bmatrix} 0.707 \\ 1.00 \end{Bmatrix}$$

$$\begin{Bmatrix} \theta_1 \\ \theta_2 \end{Bmatrix}_2 = \begin{Bmatrix} -0.707 \\ 1.00 \end{Bmatrix}$$



5-11



$$\theta_1 \approx y_1/l$$

$$\theta_2 \approx y_2/l$$

$$m_1 \ddot{y}_1 = -T \frac{y_1}{l} + T \left(\frac{y_2 - y_1}{l} \right)$$

$$m_2 \ddot{y}_2 = -T \left(\frac{y_2 - y_1}{l} \right) - T \frac{y_2}{l}$$

$$\begin{bmatrix} m_1 & 0 \\ 0 & m_2 \end{bmatrix} \begin{Bmatrix} \ddot{y}_1 \\ \ddot{y}_2 \end{Bmatrix} + \frac{T}{l} \begin{bmatrix} 2 & -1 \\ -1 & 2 \end{bmatrix} \begin{Bmatrix} y_1 \\ y_2 \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \end{Bmatrix}$$

5-12

Let $\lambda = \omega^2 \frac{m l}{T}$

in Prob 5-11

$$\begin{bmatrix} (2-\lambda) & -1 \\ -1 & (2-\lambda) \end{bmatrix} \begin{Bmatrix} Y_1 \\ Y_2 \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \end{Bmatrix}$$

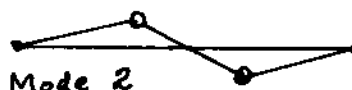
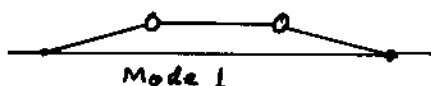
$$\lambda^2 - 4\lambda + 3 = 0$$

$$\lambda_1 = 1 \quad \omega_1 = \sqrt{T/m l}$$

$$\left(\frac{Y_1}{Y_2} \right)_1 = 1$$

$$\lambda_2 = 3 \quad \omega_2 = \sqrt{3 T/m l}$$

$$\left(\frac{Y_1}{Y_2} \right)_2 = -1$$



5-13

$$\begin{vmatrix} (2-2\lambda) & -1 \\ -1 & (2-\lambda) \end{vmatrix} = 0$$

$$2\lambda^2 - 6\lambda + 3 = 0$$

$$\lambda = \frac{3}{2} \pm \sqrt{\left(\frac{3}{2}\right)^2 - 1.5}$$

$$\lambda_1 = 0.634$$

$$\omega_1 = 0.796 \sqrt{T/m l}$$

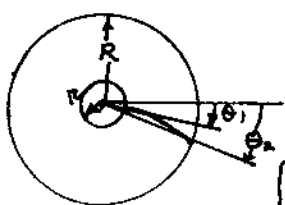
$$\begin{Bmatrix} Y_1 \\ Y_2 \end{Bmatrix}_1 = \begin{Bmatrix} 1.00 \\ 0.732 \end{Bmatrix}$$

$$\lambda_2 = 2.366$$

$$\omega_2 = 1.538 \sqrt{T/m l}$$

$$\begin{Bmatrix} Y_1 \\ Y_2 \end{Bmatrix}_2 = \begin{Bmatrix} 1.00 \\ -2.732 \end{Bmatrix}$$

5-14



$$J_1 \ddot{\theta}_1 = -K_1 \theta_1 + K_2 (\theta_2 - \theta_1)$$

$$J_2 \ddot{\theta}_2 = -K_2 (\theta_2 - \theta_1)$$

$$K_2 = 4 k_2 R^2$$

$$\text{Defl. of spring} =$$

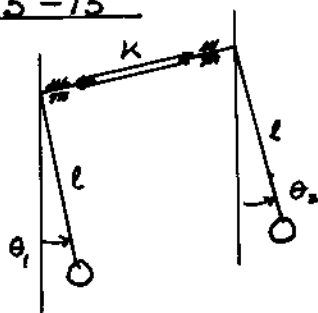
$$R(\theta_2 - \theta_1) = \frac{\text{Force}}{k_2} = \frac{F}{k_2}$$

$$\text{Torque} = R F = k_2 R^2 (\theta_2 - \theta_1)$$

$$\begin{bmatrix} \left(\frac{K_1 + K_2}{J_1} - \omega^2 \right) - \frac{K_2}{J_1} & \frac{K_2}{J_1} \\ -\frac{K_2}{J_2} & \left(\frac{K_2}{J_2} - \omega^2 \right) \end{bmatrix} \begin{Bmatrix} \theta_1 \\ \theta_2 \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \end{Bmatrix}$$

$$\omega^4 - \left[\frac{K_1}{J_1} + \frac{K_2}{J_1} + \frac{K_2}{J_2} \right] \omega^2 + \frac{K_1}{J_1} \frac{K_2}{J_2} = 0$$

5-15



$$m l^2 \ddot{\theta}_1 = -m g l \theta_1 + K (\theta_2 - \theta_1)$$

$$m l^2 \ddot{\theta}_2 = -m g l \theta_2 - K (\theta_2 - \theta_1)$$

$$\begin{bmatrix} \left(\frac{g}{l} + \frac{K}{m l^2} - \omega^2 \right) & -\frac{K}{m l^2} \\ -\frac{K}{m l^2} & \left(\frac{g}{l} + \frac{K}{m l^2} - \omega^2 \right) \end{bmatrix} \begin{Bmatrix} \theta_1 \\ \theta_2 \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \end{Bmatrix}$$

$$\omega^4 - 2 \left(\frac{g}{l} + \frac{K}{m l^2} \right) \omega^2 + \frac{g}{l} \left(\frac{g}{l} + 2 \frac{K}{m l^2} \right) = 0$$

$$\omega^2 = \frac{g}{l} + \frac{K}{m l^2} (1 \pm 1) \quad \frac{\theta_1}{\theta_2} = \frac{\frac{g}{l} + \frac{K}{m l^2} - \omega^2}{K/m l^2}$$

$$\begin{Bmatrix} \theta_1 \\ \theta_2 \end{Bmatrix}_1 = \begin{Bmatrix} 1 \\ 1 \end{Bmatrix}$$

$$\begin{Bmatrix} \theta_1 \\ \theta_2 \end{Bmatrix}_2 = \begin{Bmatrix} 1 \\ -1 \end{Bmatrix}$$

For $\left\{ \begin{array}{l} l = 19.3'' \\ m g = 3.86 \text{ lb} \\ K = 2.0 \text{ lb.in/rad} \end{array} \right\}$

$$\omega_1 = \sqrt{\frac{g}{l}} = 4.4721$$

$$\omega_2 = \sqrt{\frac{g}{l} + \frac{2K}{m l^2}} = \sqrt{20 + 1.0739} = 4.5906$$

For $\theta_1(0) = 0, \theta_2(0) = \theta_0, \dot{\theta}_1(0) = \dot{\theta}_2(0) = 0$

$$\theta_1 = \frac{\theta_0}{2} \cos \omega_1 t - \frac{\theta_0}{2} \cos \omega_2 t = \theta_0 \sin \frac{1}{2}(\omega_2 - \omega_1)t \cdot \sin \frac{1}{2}(\omega_1 + \omega_2)t$$

$$\theta_2 = \frac{\theta_0}{2} \cos \omega_1 t + \frac{\theta_0}{2} \cos \omega_2 t = \theta_0 \cos \frac{1}{2}(\omega_2 - \omega_1)t \cdot \cos \frac{1}{2}(\omega_1 + \omega_2)t$$

$$\left. \begin{array}{l} \theta_1 = \theta_0 \sin 0.0593 t \cdot \sin 4.5314 t \\ \theta_2 = \theta_0 \cos 0.0593 t \cdot \cos 4.5314 t \end{array} \right\} \text{Beat period } T_b = \frac{\pi}{0.0593} = 52.978 \text{ sec.}$$

5-16

From Prob. 5-4

$$\lambda = \begin{cases} 0.5691 & (X_1/X_2)_1 = 3.4309 \\ 4.0972 & (X_1/X_2)_2 = -0.0972 \end{cases}$$

For $x_1(0) = A, x_2(0) = 0$

$$\dot{x}_1(0) = \dot{x}_2(0) = 0$$

$$x_1 = 3.4309 C_1 \cos \omega_1 t - 0.0972 C_2 \cos \omega_2 t$$

$$x_2 = 1.0000 C_1 \cos \omega_1 t + 1.0000 C_2 \cos \omega_2 t$$

i.e. Mode 1 is multiplied by C_1 and mode 2 by C_2

At $t=0$

$$A = 3.4309 C_1 - 0.0972 C_2$$

$$0 = C_1 + C_2 \quad \therefore C_1 = -C_2$$

5-16 Cont.

$$A = -3.4309 C_2 - 0.0972 C_2 = -3.5281 C_2$$

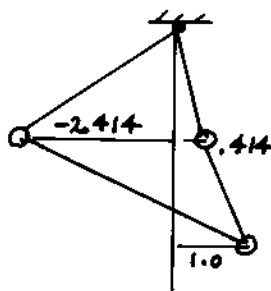
$$\therefore C_2 = -0.2834 A, \quad C_1 = 0.2834 A$$

$$x_1 = 0.9724 A \cos \omega_1 t + 0.0276 A \cos \omega_2 t$$

$$x_2 = 0.2834 A \cos \omega_1 t - 0.2834 A \cos \omega_2 t$$

5-17

Add A times mode 1 and B times mode 2



$$x_1 = 0.414 A \cos \omega_1 t - 2.414 B \cos \omega_2 t$$

$$x_2 = 1.00 A \cos \omega_1 t + 1.00 B \cos \omega_2 t$$

$$\text{At } t=0 \quad x_1(0) = x_2(0) = X$$

$$X = 0.414 A - 2.414 B$$

$$X = 1 A + 1 B \quad \therefore A = X - B$$

$$X = 0.414(X - B) - 2.414 B \quad \therefore B = -0.2072 X$$

$$A = 1.2072 X$$

$$x_1 = X \{ 0.4998 \cos \omega_1 t + 0.5002 \cos \omega_2 t \}$$

$$x_2 = X \{ 1.2072 \cos \omega_1 t - 0.2072 \cos \omega_2 t \}$$

5-18

From Prob 5-1 $(x_1/x_2)_1 = 0.614, (x_1/x_2)_2 = -1.618$

$$x_1(0) = \dot{x}_1(0) = x_2(0) = 0, \quad \dot{x}_2(0) = V$$

The 4 initial conditions require 4 arbitrary constants. We choose A times mode 1, B times mode 2 and phase angles φ_1 and φ_2

$$\begin{Bmatrix} x_1 \\ x_2 \end{Bmatrix} = A \begin{Bmatrix} 0.614 \\ 1.000 \end{Bmatrix} \cos(\omega_1 t + \varphi_1) + B \begin{Bmatrix} -1.618 \\ 1.000 \end{Bmatrix} \cos(\omega_2 t + \varphi_2)$$

$$\begin{Bmatrix} \dot{x}_1 \\ \dot{x}_2 \end{Bmatrix} = -\omega_1 A \begin{Bmatrix} 0.614 \\ 1.000 \end{Bmatrix} \sin(\omega_1 t + \varphi_1) - \omega_2 B \begin{Bmatrix} -1.618 \\ 1.000 \end{Bmatrix} \sin(\omega_2 t + \varphi_2)$$

For $t=0$

$$\begin{Bmatrix} 0 \\ 0 \end{Bmatrix} = A \begin{Bmatrix} 0.614 \\ 1.000 \end{Bmatrix} \cos \varphi_1 + B \begin{Bmatrix} -1.618 \\ 1.000 \end{Bmatrix} \cos \varphi_2$$

5-18 Cont.

$$\begin{Bmatrix} 0 \\ V \end{Bmatrix} = -\omega_1 A \begin{Bmatrix} 0.614 \\ 1.000 \end{Bmatrix} \sin \varphi_1 - \omega_2 B \begin{Bmatrix} -1.618 \\ 1.000 \end{Bmatrix} \sin \varphi_2$$

From 1st two eqs. $\varphi_1 = \varphi_2 = 90^\circ$

From 3rd eq. $\omega_1 A (0.614) = \omega_2 B (1.618)$

From 4th eq. $V = \frac{-1.618}{0.614} \omega_2 B - \omega_2 B = -3.635 \omega_2 B$

$$\therefore B = \frac{-V}{3.635 \omega_2} = -0.2751 \frac{V}{\omega_2}$$

$$A = \frac{1.618}{0.614} \frac{\omega_2}{\omega_1} \left(-0.2751 \frac{V}{\omega_2} \right) = -0.7249 \frac{V}{\omega_1}$$

$$\begin{Bmatrix} x_1 \\ x_2 \end{Bmatrix} = 0.7249 \frac{V}{\omega_1} \begin{Bmatrix} 0.614 \\ 1.000 \end{Bmatrix} \sin \omega_1 t + \frac{V}{3.635 \omega_2} \begin{Bmatrix} -1.618 \\ 1.000 \end{Bmatrix} \sin \omega_2 t$$

where: $\omega_1 = 0.618 \sqrt{\frac{k}{m}}$, $\omega_2 = 1.618 \sqrt{\frac{k}{m}}$, $\varphi_1 = \varphi_2 = 90^\circ$

5-19

Using same gen. eqs. of Prob. 5-18

$$\begin{Bmatrix} 0 \\ 1.0 \end{Bmatrix} = A \begin{Bmatrix} 0.614 \\ 1.000 \end{Bmatrix} \cos \varphi_1 + B \begin{Bmatrix} -1.618 \\ 1.000 \end{Bmatrix} \cos \varphi_2$$

$$\begin{Bmatrix} 0 \\ 0 \end{Bmatrix} = -\omega_1 A \begin{Bmatrix} 0.614 \\ 1.000 \end{Bmatrix} \sin \varphi_1 - \omega_2 B \begin{Bmatrix} -1.618 \\ 1.000 \end{Bmatrix} \sin \varphi_2$$

The second set of eqs. require $\varphi_1 = \varphi_2 = 0$

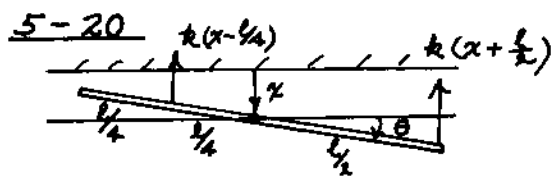
$$\therefore 0.614 A = 1.618 B \quad B = .3795 A$$

$$1.0 = A + .3795 A \quad \therefore A = .7249$$

$$B = .2751$$

Subst. into gen. eqs.

$$\begin{Bmatrix} x_1 \\ x_2 \end{Bmatrix} = \begin{Bmatrix} 0.4451 \\ 0.7249 \end{Bmatrix} \cos \omega_1 t + \begin{Bmatrix} -0.4451 \\ 0.2751 \end{Bmatrix} \cos \omega_2 t$$



$$m\ddot{x} = -k(x + \frac{l}{2}\theta) - k(x - \frac{l}{2}\theta)$$

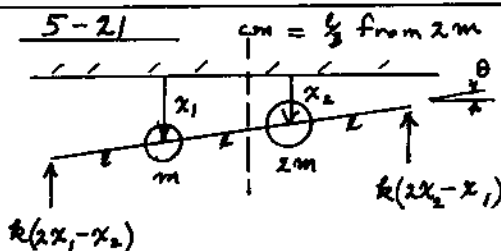
$$J\ddot{\theta} = -k(x + \frac{l}{2}\theta)\frac{l}{2} + k(x - \frac{l}{2}\theta)\frac{l}{2}$$

$$\begin{bmatrix} m & 0 \\ 0 & J \end{bmatrix} \begin{Bmatrix} \ddot{x} \\ \ddot{\theta} \end{Bmatrix} + \begin{bmatrix} 2k & k\frac{l}{4} \\ k\frac{l}{4} & k\frac{5l}{16} \end{bmatrix} \begin{Bmatrix} x \\ \theta \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \end{Bmatrix}$$

$$\begin{vmatrix} (2 - \frac{\omega^2 m}{k}) & \frac{l}{4} \\ \frac{l}{4} & (\frac{5}{16} - \frac{\omega^2 J}{k l}) \end{vmatrix} = 0$$

$$\omega^4 \left(\frac{m}{k} \frac{J}{k l} \right) - \omega^2 \left(\frac{5}{16} \frac{m}{k} + \frac{2J}{k l} \right) + \frac{9}{16} = 0$$

$$\frac{x}{\theta} = -4 \left(\frac{5}{16} - \frac{\omega^2 J}{k l} \right)$$



$$m\ddot{x}_1 + 2m\ddot{x}_2 = -k(2x_1 - x_2) - k(2x_2 - x_1)$$

$$J_{cm} = m\left(\frac{2}{3}l\right)^2 + 2m\left(\frac{l}{3}\right)^2 = \frac{2}{3}ml^2$$

$$\theta = \frac{x_1 - x_2}{l}$$

Moment eq. $\frac{2}{3}ml^2 \left(\frac{\ddot{x}_1 - \ddot{x}_2}{l} \right) = k(2x_2 - x_1)\frac{4l}{3} - k(2x_1 - x_2)\frac{5l}{3}$

$$m \begin{bmatrix} 1 & 2 \\ 2 & -2 \end{bmatrix} \begin{Bmatrix} \ddot{x}_1 \\ \ddot{x}_2 \end{Bmatrix} + k \begin{bmatrix} 1 & 1 \\ 14 & -13 \end{bmatrix} \begin{Bmatrix} x_1 \\ x_2 \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \end{Bmatrix} \quad \text{Let } \lambda = \frac{\omega^2 m}{k}$$

$$\begin{vmatrix} (1-\lambda) & (1-2\lambda) \\ (14-2\lambda) & -(13-2\lambda) \end{vmatrix} = 0$$

$$2\lambda^2 - 15\lambda + 9 = 0$$

$$\lambda_1 = 0.658 \quad \omega_1 = 0.811 \sqrt{\frac{k}{m}}$$

$$\lambda_2 = 6.842 \quad \omega_2 = 2.62 \sqrt{\frac{k}{m}}$$

$$\frac{x_1}{x_2} = \frac{(1-2\lambda)}{-(1-\lambda)}$$

$$\begin{Bmatrix} x_1 \\ x_2 \end{Bmatrix}_1 = \begin{Bmatrix} 0.921 \\ 1.00 \end{Bmatrix}$$

$$\begin{Bmatrix} x_1 \\ x_2 \end{Bmatrix}_2 = \begin{Bmatrix} -2.17 \\ 1.00 \end{Bmatrix}$$

5-22. $2m(\ddot{x}_1 - l\ddot{\theta}) + m\ddot{x}_1 = -k(x_1 + l\theta) - k(x_1 - 2l\theta)$

$$J_{cm}\ddot{\theta} = k(x_1 - 2l\theta)\frac{4l}{3} - k(x_1 + l\theta)\frac{5l}{3}$$

Both static & dynamic coupling present.

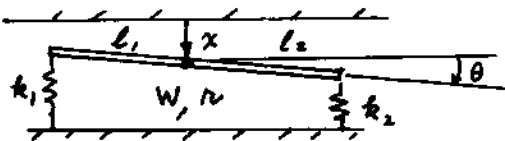
5-23 From Prob 5-9

$$\begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix} \begin{Bmatrix} \ddot{x}_1 \\ \ddot{x}_2 \end{Bmatrix} + \frac{g}{L} \begin{bmatrix} 3 & -1 \\ -1 & 1 \end{bmatrix} \begin{Bmatrix} x_1 \\ x_2 \end{Bmatrix} = 0 \quad \therefore \text{Static coupling}$$

From Prob 5-10

$$\begin{bmatrix} 1 & 0 \\ 1 & 1 \end{bmatrix} \begin{Bmatrix} \ddot{\theta}_1 \\ \ddot{\theta}_2 \end{Bmatrix} + \frac{g}{L} \begin{bmatrix} 2 & -1 \\ 0 & 1 \end{bmatrix} \begin{Bmatrix} \theta_1 \\ \theta_2 \end{Bmatrix} = 0 \quad \therefore \text{Static \& dynamic coupling}$$

5-24



$$\text{Let } a = \frac{1}{m} (k_1 + k_2)$$

$$b = \frac{1}{m} (k_2 l_2 - k_1 l_1)$$

$$c = \frac{1}{m n^2} (k_1 l_1^2 + k_2 l_2^2)$$

$$n = \text{rad. of gyr. about cm.}$$

$$\ddot{x} + ax + b\theta = 0$$

$$\ddot{\theta} + c\theta + \frac{b}{n^2}x = 0$$

$$\omega^4 - (a+c)\omega^2 + (ac - \frac{b^2}{n^2}) = 0$$

For data given

$$a = 40.5, \quad b = 42.8, \quad c = 65.6$$

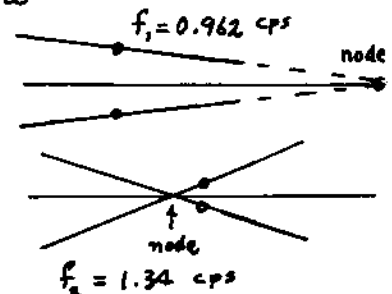
$$\omega_1^2 = 36.57$$

$$\omega_2^2 = 69.53$$

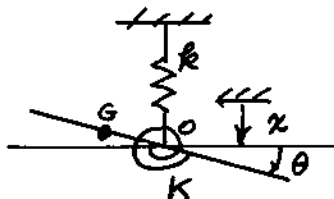
$$\left(\frac{X}{\theta}\right)_1 = -10.9 \text{ ft/rad}$$

$$\left(\frac{X}{\theta}\right)_2 = 1.47 \text{ ft/rad.}$$

$$\frac{X}{\theta} = \frac{-b}{a - \omega^2}$$



5-25



$$m(\ddot{x} - e\ddot{\theta}) + kx = 0$$

$$\sum M_G = J_G \ddot{\theta} + K\theta + kex = 0$$

$$J_0 = J_G + me^2$$

$$(J_0 - me^2)\ddot{\theta} + K\theta + kex = 0$$

5-26

$$\begin{bmatrix} m_1 & 0 \\ 0 & m_2 \end{bmatrix} \begin{Bmatrix} \ddot{x}_1 \\ \ddot{x}_2 \end{Bmatrix} + \begin{bmatrix} (k_1 + k_2) & -k_2 \\ -k_2 & k_2 \end{bmatrix} \begin{Bmatrix} x_1 \\ x_2 \end{Bmatrix} = \begin{Bmatrix} F_0 \\ 0 \end{Bmatrix} \sin \omega t$$

$$\begin{bmatrix} 0.01 & 0 \\ 0 & 0.005 \end{bmatrix} \begin{Bmatrix} \ddot{x}_1 \\ \ddot{x}_2 \end{Bmatrix} + \begin{bmatrix} 30 & -10 \\ -10 & 10 \end{bmatrix} \begin{Bmatrix} x_1 \\ x_2 \end{Bmatrix} = \begin{Bmatrix} F_0 \\ 0 \end{Bmatrix} \sin \omega t$$

5-26 Cont.

$$\begin{bmatrix} (30 - 0.01\omega^2) & -10 \\ -10 & (10 - 0.005\omega^2) \end{bmatrix} \begin{Bmatrix} X_1 \\ X_2 \end{Bmatrix} = \begin{Bmatrix} F_0 \\ 0 \end{Bmatrix}$$

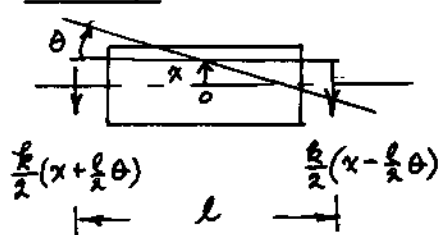
$$\omega^4 - 5000\omega^2 + 4 \times 10^6 = 0$$

$$\omega^2 = 2500 \pm \sqrt{2500^2 - 4 \times 10^6} = 2500 \pm \sqrt{2.25 \times 10^6}$$

$$\omega_1 = 31.6 \text{ rad/s} \quad \left(\frac{X_1}{X_2}\right)_1 = \frac{1}{2}$$

$$\omega_2 = 63.3 \text{ rad/s} \quad \left(\frac{X_1}{X_2}\right)_2 = -1.0$$

5-27

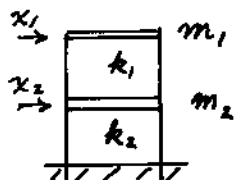


$$\Sigma F = -kx + m\omega^2 \cos \omega t = (M+m)\ddot{x} \approx M\ddot{x}$$

$$\Sigma M_o = -\frac{k}{2}l^2\theta + m\omega^2 b \cos \omega t = J_o \ddot{\theta}$$

$$m \ll M$$

5-28



$$m_1 \ddot{x}_1 + k_1(x_1 - x_2) = 0$$

$$m_2 \ddot{x}_2 - k_1(x_1 - x_2) + k_2 x_2 = 0$$

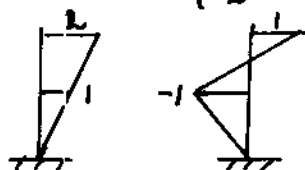
$$\text{Let } \lambda = \frac{\omega^2 m_1}{k_1}, \quad m_2 = 2m_1, \quad k_2 = 2k_1$$

$$(1-\lambda)x_1 - x_2 = 0 \quad \left. \begin{array}{l} \lambda^2 - \frac{5}{2}\lambda + 1 = 0 \end{array} \right\}$$

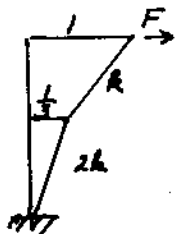
$$-x_1 + (3-2\lambda)x_2 = 0$$

$$\lambda = \begin{cases} \frac{1}{2} \\ 2 \end{cases} \quad \omega_1 = \sqrt{\frac{k_1}{2m_1}}, \quad \omega_2 = \sqrt{\frac{2k_1}{m_1}}$$

$$\frac{X_1}{X_2} = \frac{1}{1-\lambda} = \begin{cases} 2 \\ -1 \end{cases}$$



5-29



Initial Displ.

$$\begin{Bmatrix} x_1 \\ x_2 \end{Bmatrix} = \begin{Bmatrix} 2 \\ 1 \end{Bmatrix} A \cos \omega_1 t + \begin{Bmatrix} -1 \\ 1 \end{Bmatrix} B \cos \omega_2 t$$

Satisfies $\dot{x}(0) = \dot{x}_2(0) = 0$

$$\begin{Bmatrix} 1 \\ 1/3 \end{Bmatrix} = \begin{Bmatrix} 2 \\ 1 \end{Bmatrix} A + \begin{Bmatrix} -1 \\ 1 \end{Bmatrix} B \quad \therefore A = \frac{4}{9}, \quad B = -\frac{1}{9}$$

$$\begin{Bmatrix} x_1 \\ x_2 \end{Bmatrix} = \begin{Bmatrix} 8/9 \\ 4/9 \end{Bmatrix} \cos \omega_1 t + \begin{Bmatrix} 1/9 \\ -1/9 \end{Bmatrix} \cos \omega_2 t$$

5-30 From Prob 5-29

$$x_1 = \frac{8}{9} \cos \omega_1 t + \frac{1}{9} \cos \omega_2 t$$

$$\omega_1 = \sqrt{\frac{k_1}{2m_1}}$$

$$x_2 = \frac{4}{9} \cos \omega_1 t - \frac{1}{9} \cos \omega_2 t$$

$$\omega_2 = \sqrt{\frac{2k_1}{m_1}} = 2\omega_1$$

$$\left. \begin{array}{l} \text{shear in 1st story} = k_2 x_2 = 2k_1 x_2 \\ \text{" " 2nd " " } = k_1 (x_1 - x_2) \end{array} \right\} \text{Ratio} = \frac{2x_{2\max}}{(x_1 - x_2)_{\max}}$$

$$\frac{\partial x_2}{\partial t} = -\frac{4}{9} \omega_1 \sin \omega_1 t + \frac{1}{9} \omega_2 \sin \omega_2 t = 0$$

$$= -\frac{\omega_1}{9} \{ 4 \sin \omega_1 t - 2 \sin 2\omega_1 t \} = 0$$

$$= \{ 2 \sin \omega_1 t - 2 \sin \omega_1 t \cos \omega_1 t \} = 0$$

$$\therefore \cos \omega_1 t = 1, \text{ and } \omega_1 t = 0, 360^\circ \therefore x_{2\max} = \frac{4}{9} - \frac{1}{9} = \frac{1}{3}$$

$$\frac{\partial (x_1 - x_2)}{\partial t} = -\frac{4}{9} \omega_1 \sin \omega_1 t - \frac{1}{9} \omega_2 \sin 2\omega_1 t = 0$$

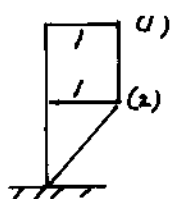
$$\text{or } \sin \omega_1 t (1 + 2 \cos \omega_1 t) = 0, \quad \cos \omega_1 t = -\frac{1}{2}, \quad \omega_1 t = 120^\circ$$

$$\therefore (x_1 - x_2)_{\max} = \frac{4}{9} \left(-\frac{1}{2}\right) + \frac{1}{9} \left(-\frac{1}{2}\right) = \frac{1}{3}$$

$$\text{Ratio of shears} = \frac{2(\frac{1}{3})}{(\frac{1}{3})} = 2$$

5-31

See Prob. 5-29



$$\begin{Bmatrix} 1 \\ 1 \end{Bmatrix} = \begin{Bmatrix} 2 \\ 1 \end{Bmatrix} A + \begin{Bmatrix} -1 \\ 1 \end{Bmatrix} B$$

$$A = \frac{2}{3}, \quad B = \frac{1}{3}$$

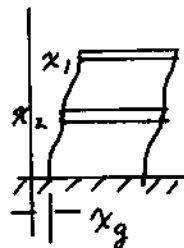
$$\begin{Bmatrix} x_1 \\ x_2 \end{Bmatrix} = \begin{Bmatrix} 4/3 \\ 2/3 \end{Bmatrix} \cos \omega_1 t + \begin{Bmatrix} -1/3 \\ 1/3 \end{Bmatrix} \cos \omega_2 t$$

5-32

see Prob. 5-28

with $\lambda = \frac{\omega^2 m_1}{k_1}$

$$\begin{bmatrix} (1-\lambda) & -1 \\ -1 & (3-2\lambda) \end{bmatrix} \begin{Bmatrix} x_1 \\ x_2 \end{Bmatrix} = \begin{Bmatrix} 0 \\ 2x_g \end{Bmatrix}$$



5-32 Cont.

$$\lambda_1 = \frac{1}{2} \quad \lambda_2 = 2$$

\therefore Denominator $= (\lambda - \frac{1}{2})(\lambda - 2)$ use Cramer's rule

$$X_1 = \frac{\begin{vmatrix} 0 & -1 \\ 2 & (3-2\lambda) \end{vmatrix} X_g}{(\lambda - \frac{1}{2})(\lambda - 2)}$$

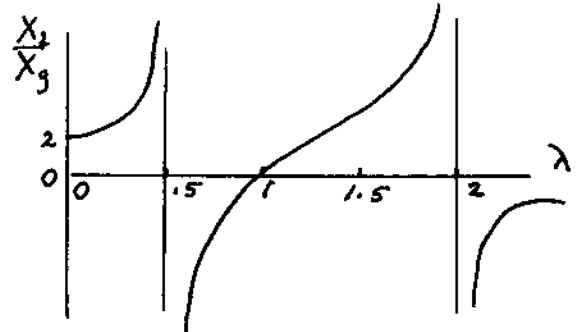
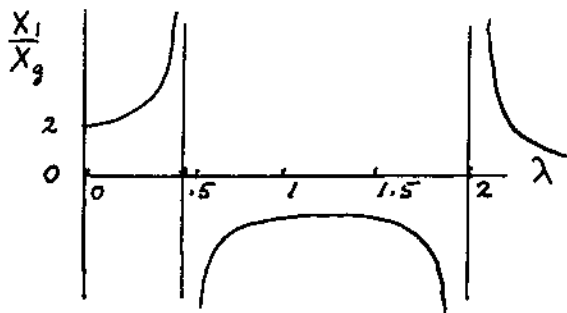
$$X_2 = \frac{\begin{vmatrix} (1-\lambda) & 0 \\ -1 & 2 \end{vmatrix} X_g}{(\lambda - \frac{1}{2})(\lambda - 2)}$$

$$\text{Response} = x_1 = X_1 \sin \omega t$$

$$x_2 = X_2 \sin \omega t$$

$$\frac{x_1}{X_g} = \frac{2 \sin \omega t}{(\lambda - \frac{1}{2})(\lambda - 2)}$$

$$\frac{x_2}{X_g} = \frac{2(1-\lambda) \sin \omega t}{(\lambda - \frac{1}{2})(\lambda - 2)}$$



5-33

$$M(\ddot{y}_0 - l_0 \ddot{\theta}) = K_R(y_G - y_0)$$

$$M \rho_c^2 \ddot{\theta} = K_R(y_G - y_0) l_0 - K_R \theta + M g l_0 \theta$$

ρ_c = rad. of gyration of bldg about its c.m.

5-34

$$\text{Let } \omega_h^2 = \frac{K_h}{M}$$

$$\omega_n^2 = \frac{K_R}{M \rho_c^2}$$

$$\lambda = \frac{\omega}{\omega_h}, \quad \left(\frac{\rho_c}{l_0}\right)^2 = \frac{1}{3}, \quad \left(\frac{\omega_n}{\omega_h}\right)^2 = 4$$

$$\begin{bmatrix} (1-\lambda^2) & \lambda^2 \\ 1 & \frac{1}{3}(4-\lambda^2) \end{bmatrix} \begin{Bmatrix} Y_0 \\ l_0 \theta_0 \end{Bmatrix} = Y_G e^{i\phi} \begin{Bmatrix} 1 \\ 1 \end{Bmatrix}$$

Characteristic eq.

$$\lambda^4 - 8\lambda^2 + 4 = 0$$

$$\lambda^2 = 4 \pm \sqrt{12} = \begin{cases} 0.54 \\ 7.46 \end{cases}$$

$$\lambda = \frac{\omega}{\omega_h} = \begin{cases} 0.734 \\ 2.73 \end{cases}$$

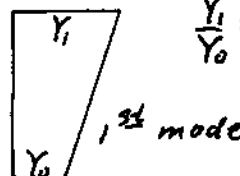
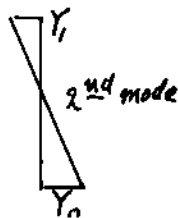
Ampl. ratio

$$\text{1st mode } \frac{Y_0}{l_0 \theta_0} = \frac{\lambda^2 - 4}{3} = -1.15$$

$$\text{Let } Y_1 = Y_0 - 2l_0 \theta_0 = \text{displ of top, then } \frac{Y_1}{l_0 \theta_0} = \frac{Y_0}{l_0 \theta_0} - 2 = -1.15 - 2 = -3.15$$

2nd mode

$$\left. \begin{aligned} \frac{Y_0}{l_0 \theta_0} &= 1.15 \\ \frac{Y_1}{l_0 \theta_0} &= -0.85 \end{aligned} \right\} \frac{Y_1}{Y_0} = -0.74$$



$$\frac{Y_1}{Y_0} = \frac{-3.15}{-1.15} = 2.73$$

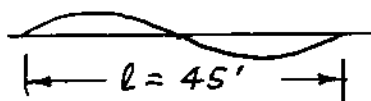
5-35 From Prob. 5-34 $\text{Det.} = \lambda^4 - 8\lambda^2 + 4$

For $\text{Det} = 0$, $\lambda = \frac{\omega}{\omega_n} = \begin{cases} 0.732 & 1^{\text{st}} \text{ mode} \\ 2.732 & 2^{\text{nd}} \text{ mode} \end{cases}$

Using Cramer's rule

$$\left. \begin{aligned} \frac{Y_0}{Y_G} &= \frac{\begin{vmatrix} 1 & \lambda^2 \\ 1 & \frac{1}{3}(4-\lambda^2) \end{vmatrix}}{\lambda^4 - 8\lambda^2 + 4} = \frac{\frac{4}{3}}{\lambda^4 - 8\lambda^2 + 4} \frac{(1-\lambda^2)}{\lambda^4 - 8\lambda^2 + 4} \\ \frac{L_0 \theta_0}{Y_G} &= \frac{\begin{vmatrix} (1-\lambda^2) & 1 \\ 1 & 1 \end{vmatrix}}{\lambda^4 - 8\lambda^2 + 4} = \frac{-\lambda^2}{\lambda^4 - 8\lambda^2 + 4} \end{aligned} \right\} \begin{array}{l} \text{Plot for various} \\ \text{values of } \lambda \text{ to} \\ \text{Check Fig P5-35} \end{array}$$

5-36



Unfavorable speed is found from $v \gamma = l$

$$v = \frac{l}{\gamma} = l f_n$$

From Prob 5-24

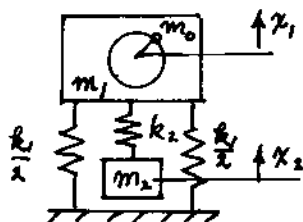
$f_1 = 0.962$ predominately bouncing

$f_2 = 1.327$ " pitching

$v_1 = 45 \times 0.962 = 43.29 \text{ ft/s} = 29.5 \text{ mph}$

$v_2 = 45 \times 1.327 = 59.72 \text{ ft/s} = 40.7 \text{ mph}$

5-37



$$m_1 \ddot{x}_1 + k_1 x_1 + k_2 (x_1 - x_2) = m_0 e \omega^2 \sin \omega t$$

$$m_2 \ddot{x}_2 + k_2 (x_2 - x_1) = 0$$

$\frac{k_2}{m_2}$ should equal exciting freq ω^2

$$k_2 = \omega^2 m_2 = \left(\frac{2\pi 1800}{60} \right)^2 \left(\frac{50}{386} \right) = 4600 \text{ lb/in}$$

The absorber will then make $x_1 = 0$ and the absorber force will be equal and opposite to the exciting force.

$$k_2 x_2 = m_0 e \omega^2$$

$$x_2 = \frac{m_0 e \omega^2}{k_2} = \frac{m_0 e}{m_2} = \frac{2}{50} = 0.04''$$

5-38

$$m_1 \ddot{x}_1 + c(\dot{x}_1 - \dot{x}_2) + k_1 x_1 + k_2(x_1 - x_2) = m_1 \omega^2 (e^{i\omega t})$$

$$m_2 \ddot{x}_2 + c(\dot{x}_2 - \dot{x}_1) + k_2(x_2 - x_1) = 0$$

Let $x_1 = X_1 e^{i\omega t}$, $x_2 = X_2 e^{i\omega t}$ where X_1, X_2 are complex

$$\left(\frac{k_1 + k_2}{m_1} - \omega^2 + i \frac{\omega c}{m_1}\right) X_1 - \left(\frac{k_2}{m_1} + \frac{i\omega c}{m_1}\right) X_2 = \frac{(m_1 \omega^2)}{m_1}$$

$$-\left(\frac{k_2}{m_2} + \frac{i\omega c}{m_2}\right) X_1 + \left(\frac{k_2}{m_2} - \omega^2 + \frac{i\omega c}{m_2}\right) X_2 = 0$$

$$X_1 = \frac{(m_1 \omega^2) (k_2 - m_2 \omega^2 + i\omega c)}{[(k_1 + k_2) - m_1 \omega^2 + i\omega c][k_2 - m_2 \omega^2 + i\omega c] - (k_2 + i\omega c)^2}$$

$$\frac{X_2}{X_1} = \frac{(k_2 - i\omega c)}{(k_1 - m_2 \omega^2 + i\omega c)}$$

5-39

$$I \ddot{\theta}_1 + 4ka^2(\theta_1 - \theta_2) = M_0 \sin \omega t$$

$$I_2 \ddot{\theta}_2 + 4ka^2(\theta_2 - \theta_1) = 0$$

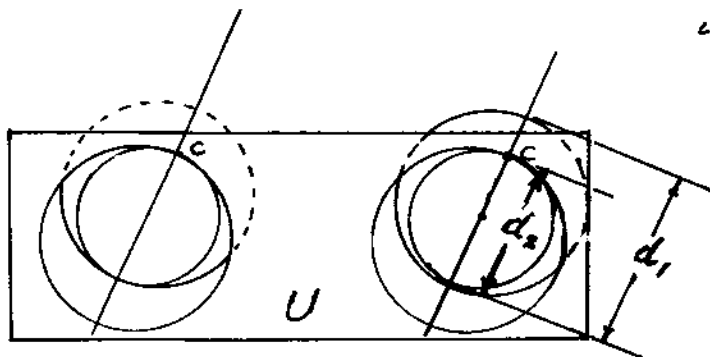
Let $\lambda = \frac{\omega^2 I}{ka^2}$ and $n = \frac{I_2}{I}$

$$\begin{bmatrix} (4-\lambda) & -1 \\ -1 & (4-n\lambda) \end{bmatrix} \begin{Bmatrix} \theta_1 \\ \theta_2 \end{Bmatrix} = \begin{Bmatrix} M_0 \\ 0 \end{Bmatrix} \sin \omega t$$

$$\theta_1 = \frac{(4-n\lambda) M_0 \sin \omega t}{n\lambda^2 - 4(1+n)\lambda + 15} \quad \therefore \theta_1 = 0 \text{ when } \lambda = \frac{4}{n}$$

5-40

Point of contact c of U with small circle moves in circle of radius $r = d_1 - d_2$



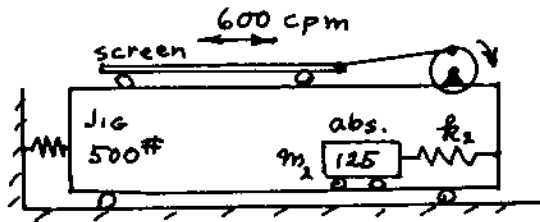
5-41 From Eq. 5.6-5 the natural freq. is

$$\omega_n = n \sqrt{\frac{R}{r}} = 4 (\text{rot. speed}) = 4n$$

$$\therefore \frac{n}{R} = \frac{1}{16} \quad r = d_1 - d_2 = \frac{3}{4}'' - d_2$$

$$\therefore r = \frac{R}{16} = \frac{4''}{16} = \frac{1}{4}'' = \frac{3}{4}'' - d_2 \quad d_2 = \frac{1}{2}''$$

5-42



$$f_1 = 400 \text{ cpm.}$$

$$\text{Excit. } \omega = \frac{2\pi 600}{60} = 20\pi \text{ rad/s.}$$

Nat. freq. of absorber must equal the excitation freq.

$$\omega_{22}^2 = \frac{k_2}{m_2} = \frac{386 k_2}{125} = (20\pi)^2$$

$$\therefore k_2 = 1278. \text{ \#/in}$$

Nat. freq. of system is found from the denominator of Eq 5.5-1 which can be reduced to

$$\left(\frac{\omega}{\omega_{22}}\right)^4 - \left[1 + \left(\frac{\omega_{11}}{\omega_{22}}\right)^2 \left\{1 + \mu \left(\frac{\omega_{22}}{\omega_{11}}\right)^2\right\}\right] \left(\frac{\omega}{\omega_{22}}\right)^2 + \left(\frac{\omega_{11}}{\omega_{22}}\right)^2 = 0$$

$$\mu = \frac{m_2}{m_1} = \frac{125}{500} = 0.25, \quad \left(\frac{\omega_{11}}{\omega_{22}}\right)^2 = \left(\frac{400}{600}\right)^2 = \frac{1}{2.25}, \quad \text{Let } \lambda = \frac{\omega}{\omega_{22}}$$

$$\lambda^4 - 1.695 \lambda^2 + \frac{1}{2.25} = 0, \quad \lambda^2 = \left(\frac{\omega}{\omega_{22}}\right)^2 = 0.845 \pm 0.164$$

5-43 Refer to Fig 5.5-3

With trial weight of 2 lb tuned to 232 rpm, the two nat. freqs. are

$$\left(\frac{\omega}{\omega_{22}}\right) = \frac{198}{232} = 0.854 \quad \text{and} \quad \left(\frac{\omega}{\omega_{22}}\right) = \frac{272}{232} = 1.17$$

These numbers establish the mass ratio from Fig 5.5-3 to be $\mu \approx 0.10$

To move nat. freqs. outside specified freqs. of

$$\left(\frac{\omega}{\omega_{22}}\right) = \frac{160}{232} = 0.69 \quad \text{and} \quad \left(\frac{\omega}{\omega_{22}}\right) = \frac{320}{232} = 1.38$$

Fig 5.5-3 shows $\mu \geq 0.57$

5-43 Cont.

Since $\mu_1 = \frac{(m_2)_1}{m_1} = \frac{2}{m_1} = 0.10 \quad \therefore m_1 = 20$

$\mu_2 = \frac{(m_2)_2}{m_1} = 0.57, \quad (m_2)_2 = .57 \times 20 = 11.4 \text{ lb.}$

The stiffness should be $k_2 = m_2 \omega^2 = \frac{11.4}{386} \left(\frac{2\pi \cdot 232}{60} \right)^2 = 17.9 \text{ #/in}$

5-44 Assume linear velocity distribution of fluid between disk and case. The torque is

$$\begin{aligned} T &= \mu (\text{velocity gradient})(\text{radius})(\text{area}) \\ &= 2 \int_{R_0}^R 2\pi \mu \left(\frac{\omega r}{h} \right) r^2 dr + 2\pi \mu \left(\frac{\omega R}{h} \right) R^2 b \\ &= 2\pi \mu \frac{\omega R^3}{h} \left[\frac{1}{2} \left(R - \frac{R_0^4}{R^3} \right) + b \right] \end{aligned}$$

5-45 Optimum damping given by Eq. 5.7-6

$$\zeta_0 = \frac{\mu}{\sqrt{2(1+\mu)(2+\mu)}} = \frac{.25}{\sqrt{2(1.25)(2.25)}} = 0.1054$$

The freq. at which the damper is most effective (with peak ampl.) is given by Eq. 5.7-7

$$\frac{\omega}{\omega_n} = \sqrt{\frac{2}{2+\mu}} = \sqrt{\frac{2}{2.25}} = 0.943$$

5-46 The peak amplitude for any μ and ζ can be found from Fig 5.7-4. It is seen that the optimum (lowest point on curve) for $\mu = .25$ is $\zeta \approx .105$ as computed in Prob. 5-45. Thus

$$\frac{\theta_{\max} \zeta = .10}{\theta_{\max} \zeta = .105} \approx 1.0$$

5-47 In Fig 5.7-3 all curves pass through a common point p . This point is at the frequency for optimum damping. Thus by equating $|\frac{K\theta_0}{M_0}|^2$ for $\zeta=0$ and $\zeta=\infty$ Eq. 5.7-7 is obtained

$$\frac{\mu^2(\omega/\omega_n)^2}{\mu^2(\omega/\omega_n)^2(1-\omega^2/\omega_n^2)} = \frac{4}{4[\mu(\frac{\omega}{\omega_n})^2 - (1-\omega^2/\omega_n^2)]^2}$$

or.
$$\frac{1}{1-(\omega/\omega_n)^2} = \frac{1}{(\frac{\omega}{\omega_n})^2(1+\mu)-1}$$

Thus
$$\omega/\omega_n = \sqrt{\frac{2}{2+\mu}}$$

Eq. 5.7-6 is found by differentiating $|\frac{K\theta_0}{M_0}|^2$ wrt. $(\frac{\omega}{\omega_n})^2$, equating it to zero and substituting Eq. 5.7-7

Let $n^2 = \frac{2}{2+\mu} = (\frac{\omega}{\omega_n})^2$ $(1-n^2) = \frac{\mu}{2+\mu} = n^2(1+\mu) - 1$

$q = \frac{4\zeta^2}{\mu^2}$ Rewrite Eq. 5.7-5

$$|\frac{K\theta_0}{M_0}|^2 = \frac{n^2+q}{n^2(1-n^2)^2 + q[n^2(1+\mu)-1]^2}$$

$$\frac{\partial ||^2}{\partial n^2} = \frac{n^2(1-n^2)^2 + q[n^2(1+\mu)-1]^2 - (n^2+q)\{(1-n^2)^2 - 2n^2(1-n^2) + 2q[n^2(1+\mu)-1](1+\mu)\}}{[n^2(1-n^2)^2 + q[n^2(1+\mu)-1]^2]^2} = 0$$

$$= n^2(1-n^2)^2 + q(1-n^2)^2 - (n^2+q)\{ \text{same} \} = 0$$

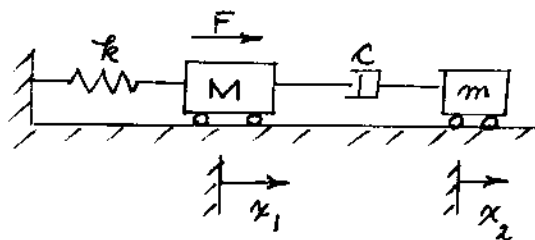
$$= (n^2+q)\left[\frac{\mu}{2+\mu} - \frac{\mu}{2+\mu} + 2n^2\left(\frac{\mu}{2+\mu}\right) - 2q\left(\frac{\mu}{2+\mu}\right)(1+\mu)\right] = 0$$

$$\therefore n^2 - q(1+\mu) = 0 \text{ and } q = \frac{n^2}{1+\mu} = \frac{4\zeta^2}{\mu^2}$$

$$\zeta^2 = \frac{\mu^2}{4} \cdot \frac{1}{1+\mu} \cdot \frac{2}{2+\mu} = \frac{\mu^2}{2(1+\mu)(2+\mu)}$$

$$\zeta_{opt} = \frac{\mu}{\sqrt{2(1+\mu)(2+\mu)}}$$

5-48



$$M \ddot{x}_1 = -kx_1 - c(\dot{x}_1 - \dot{x}_2) + F e^{i\omega t}$$

$$m \ddot{x}_2 = c(\dot{x}_1 - \dot{x}_2) \quad \text{Let } x_1 = X_1 e^{i\omega t}, \quad x_2 = X_2 e^{i\omega t}$$

$$\left[\left(\frac{k}{M} - \omega^2 \right) + i \left(\frac{c\omega}{M} \right) \right] X_1 - i \left[\frac{c\omega}{M} \right] X_2 = \frac{F}{M}$$

$$\left[-\omega^2 + i \frac{c\omega}{m} \right] X_2 = i \left(\frac{c\omega}{m} \right) X_1 \quad \text{eliminate } X_2$$

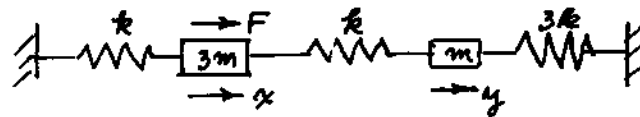
$$\left\{ \left[\left(\frac{k}{M} - \omega^2 \right) + i \left(\frac{c\omega}{M} \right) \right] - i \left(\frac{c\omega}{M} \right) \frac{i \left(\frac{c\omega}{m} \right)}{(-\omega^2 + i \frac{c\omega}{m})} \right\} X_1 = \frac{F}{M}$$

$$\left\{ \left[(k - M\omega^2) + ic\omega \right] \left(\frac{-m\omega^2 + ic\omega}{\cancel{m}} \right) + \frac{(c\omega)^2}{\cancel{m}} \right\} X_1 =$$

$$F \left(\frac{-m\omega^2 + ic\omega}{\cancel{m}} \right)$$

$$\therefore \frac{X_1}{F} = \frac{(\omega^2 m - ic\omega)}{m\omega^2(k - M\omega^2) + ic\omega[m\omega^2 - (k - M\omega^2)]}$$

5-49



$$\left. \begin{aligned} \ddot{x} &= -\frac{2}{3} \frac{k}{m} x + \frac{1}{3} \frac{k}{m} y + \frac{F}{3m} \\ \ddot{y} &= \frac{k}{m} x - 4 \frac{k}{m} y \end{aligned} \right\} \begin{aligned} \omega_1 &= 0.751 \sqrt{\frac{k}{m}} \\ \omega_2 &= 2.04 \sqrt{\frac{k}{m}} \end{aligned}$$

Let $k = m = 1$, then $\gamma_1 = 8.35 \text{ sec}$, $\gamma_2 = 3.07 \text{ sec}$.

Let $\Delta t = 0.20$

$$x_{i+1} = \ddot{x}_i \Delta t^2 + 2x_i - x_{i-1}$$

$$y_{i+1} = \ddot{y}_i \Delta t^2 + 2y_i - y_{i-1}$$

Initial Cond. ($I=1$)

$$x(1) = y(1) = \dot{y}(1) = 0, \quad \ddot{x}_1(1) = \frac{100}{3}$$

From Eq. (4.5-8) $x(2) = \frac{1}{2} (0.20)^2 \left(\frac{100}{3} \right) = 0.666$

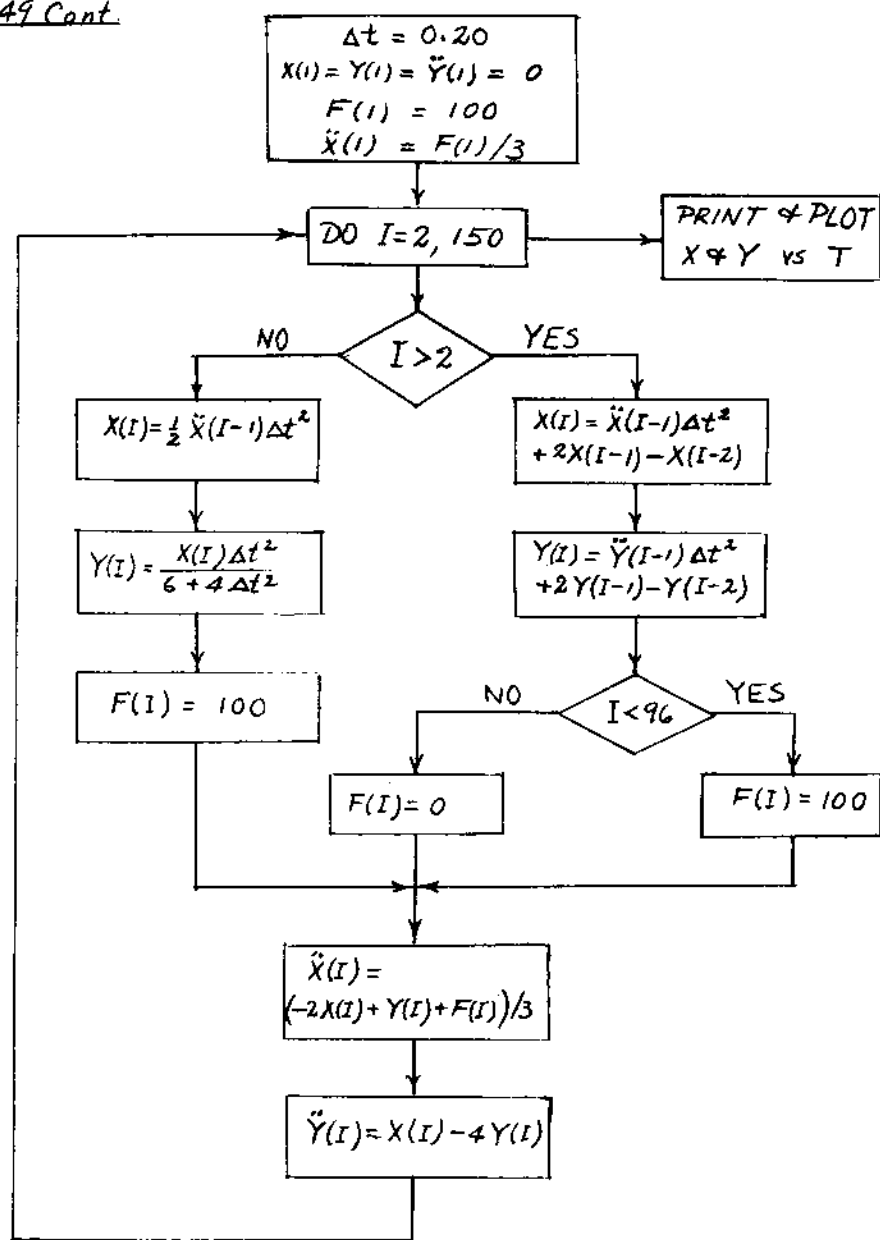
From Eq. (4.5-10) and DE for \ddot{y}

$$y(2) = \frac{1}{6} \Delta t^2 \ddot{y}(2) = \frac{\Delta t^2}{6} [x(2) - 4y(2)]$$

$$y(2) \left[1 + \frac{2}{3} \Delta t^2 \right] = \frac{1}{6} \Delta t^2 \times 0.666 \quad \therefore y(2) = 0.00432$$

Flow diagram, computer program and plot of results follow.

5-49 Cont.

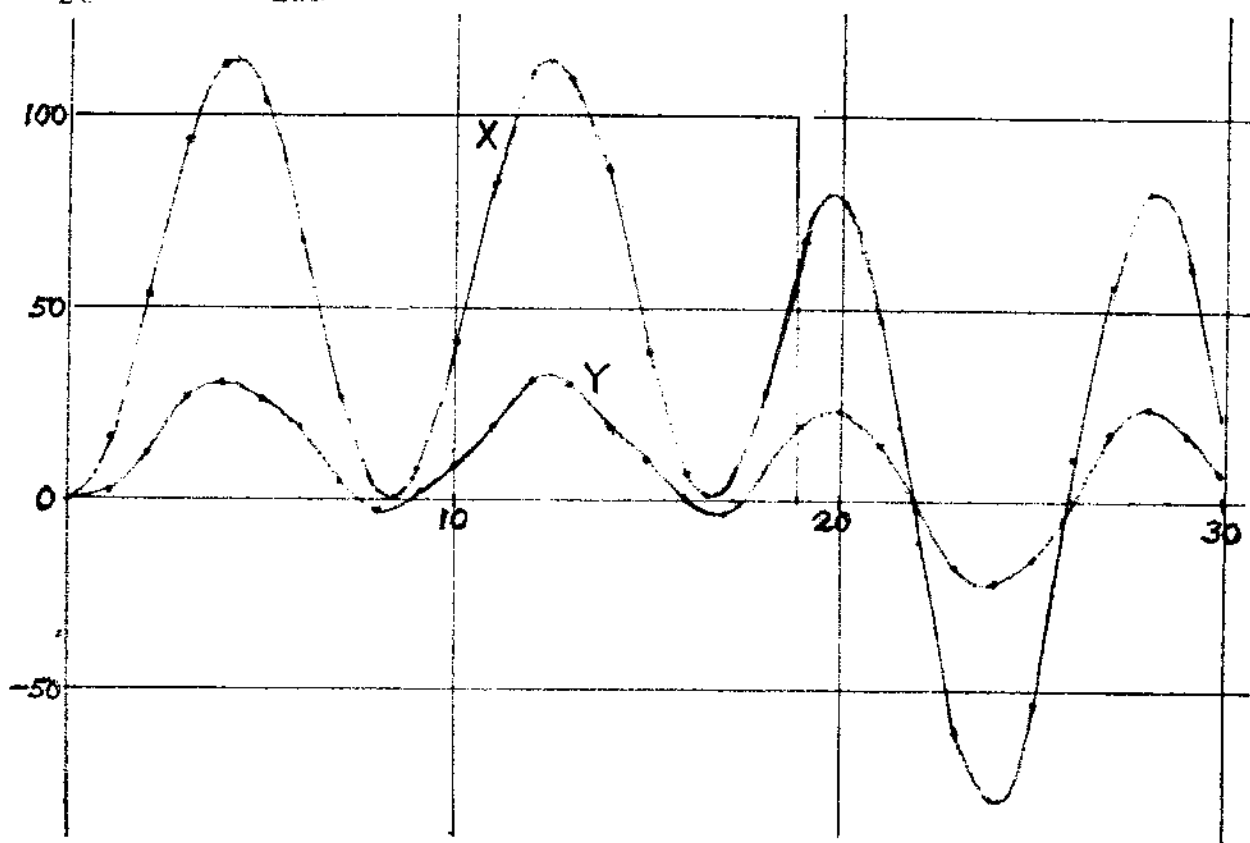


Flow Diagram Prob 5-49

Computer Program - Prob 5-49

```

C      PROBLEM 5-49 THOMSON
1      DIMENSION X(155),Y(155),DX(155),DY(155),T(155)
2      DT=0.20
3      F=100.
4      X(1)=0.0
5      Y(1)=0.0
6      DX(1)=0.0
7      DX(1)=F/3.
8      T(1)=0.0
9      I=2
10     5 IF(I.GT.2) GOTO 10
11     X(I)=0.5*DX(I-1)*DT**2
12     Y(I)=X(I)*DT**2/(6.+4.*DT**2)
13     GO TO 30
14     10 X(I)=DX(I-1)*DT**2+2.*X(I-1)-X(I-2)
15     Y(I)=DY(I-1)*DT**2+2.*Y(I-1)-Y(I-2)
16     IF(I.GT.95) GOTO 20
17     F=100.
18     GO TO 30
19     20 F=0.0
20     30 DX(I)=(-2.*X(I)+Y(I)+F)/3.
21     DY(I)=X(I)-4.*Y(I)
22     T(I)=T(I-1)+DT
23     I=I+1
24     IF(I.LE.151) GOTO 5
25     DO 50 I=1,150
26     50 PRINT,T(I),X(I),Y(I)
27     STOP
28     END
    
```



5-50

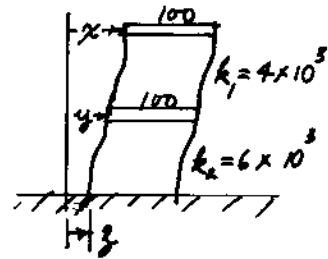
$$100 \ddot{x} = -4 \times 10^3 (x - y)$$

$$100 \ddot{y} = 4 \times 10^3 (x - y) - 6 \times 10^3 (y - z)$$

$$\therefore \ddot{x} = -40x + 40y$$

$$\ddot{y} = 40x - 100y + 60z$$

$$z = 10 \sin \pi t$$



Use Eq. 4.5-10 to start computation

$$x(z) = \frac{1}{6} \ddot{x}(z) h^2 = \frac{h^2}{6} [-40x(z) + 40y(z)]$$

$$y(z) = \frac{h^2}{6} \ddot{y}(z) = \frac{h^2}{6} [40x(z) - 100y(z) + 60z(z)]$$

$$x(z) \left[1 + \frac{40}{6} h^2 \right] = y(z) \frac{40}{6} h^2$$

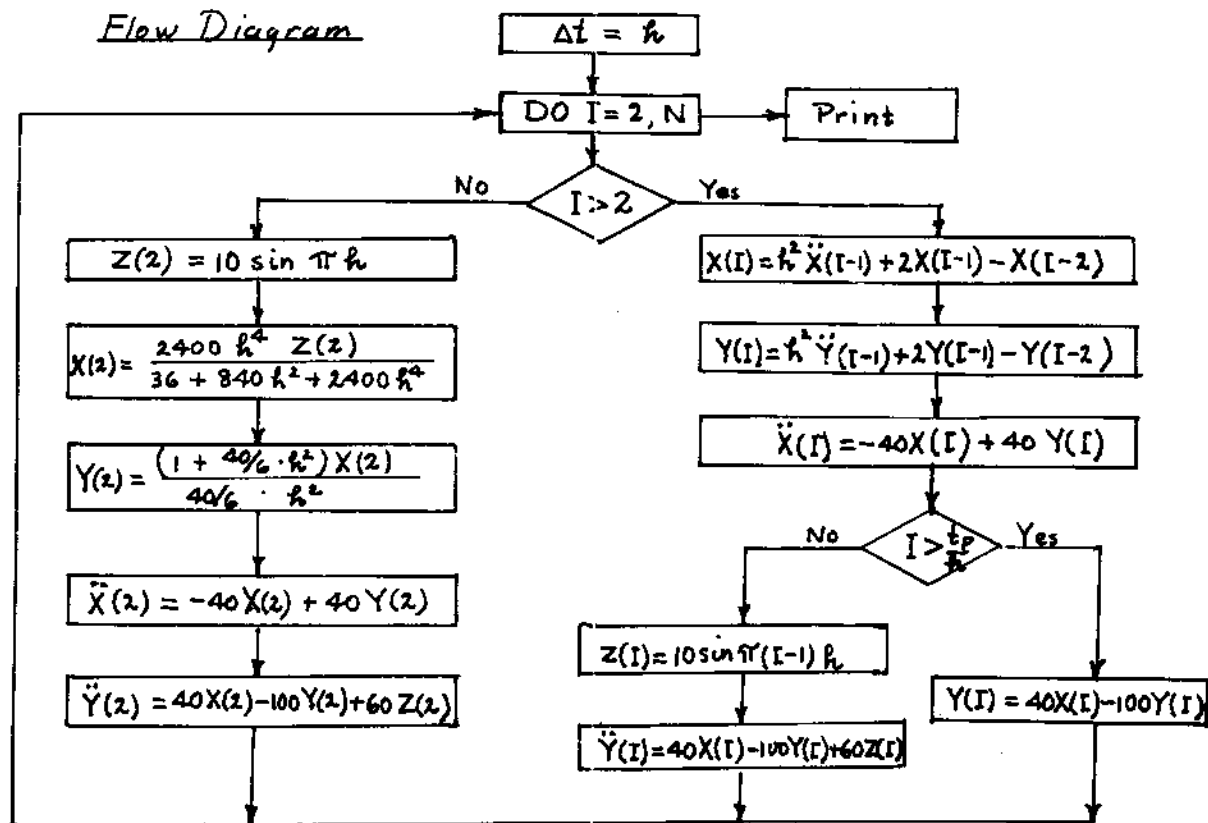
$$y(z) \left[1 + \frac{100}{6} h^2 \right] = x(z) \frac{40}{6} h^2 + 10 h^2 z(z)$$

$$\therefore x(z) = \frac{40}{6} h^2 \frac{1}{1 + \frac{40}{6} h^2} \cdot \frac{1}{1 + \frac{100}{6} h^2} \cdot [x(z) \frac{40}{6} h^2 + 10 h^2 z(z)]$$

$$x(z) = 2400 h^4 z(z) / (36 + 840 h^2 + 2400 h^4)$$

$$y(z) = (1 + \frac{40}{6} h^2) x(z) / \frac{40}{6} h^2$$

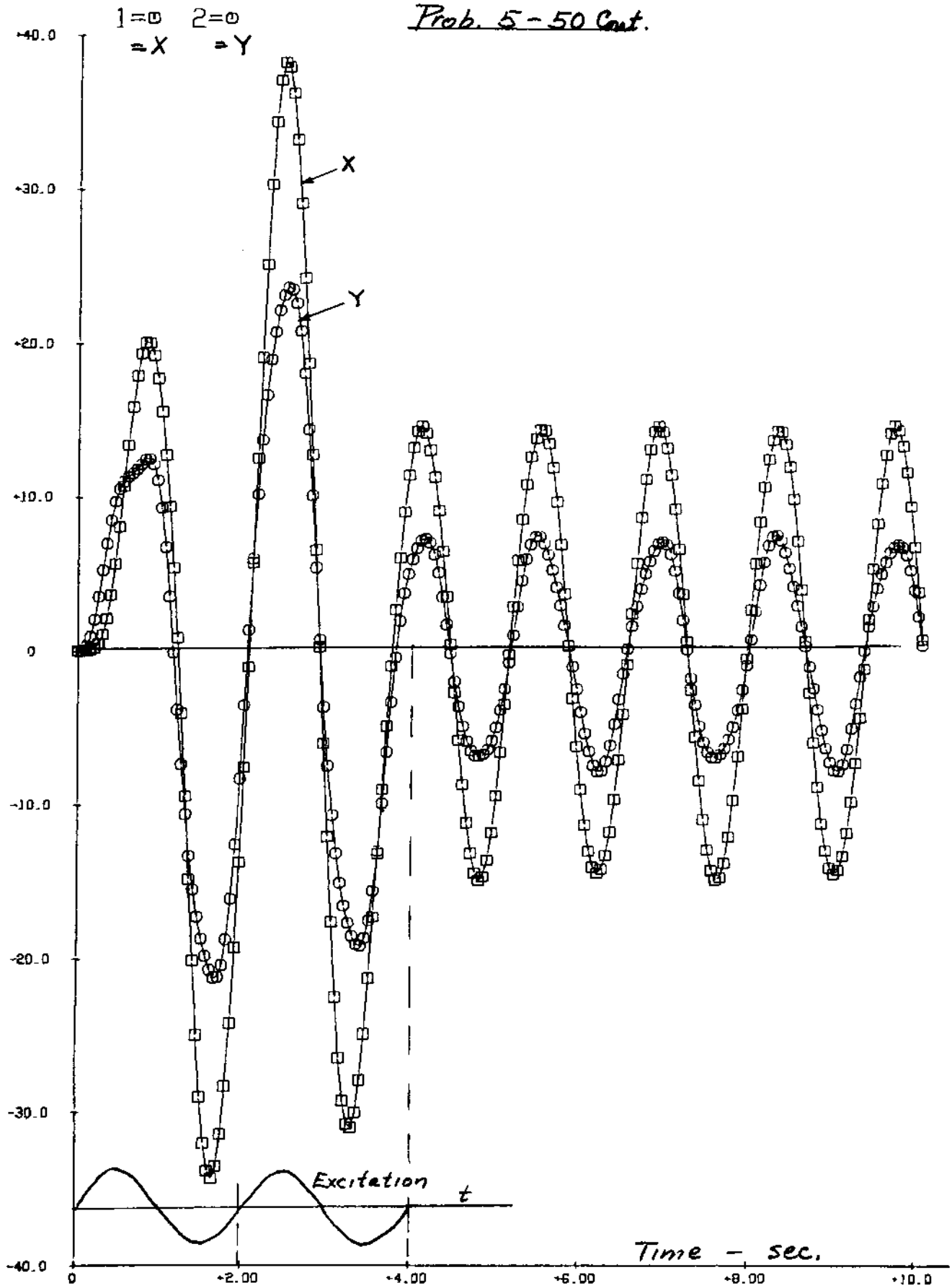
starting equations



Computer Program

```
PROBLEM 5-50 THOMSON
DIMENSION X(220),Y(220),DX(220),DY(220),T(220),Z(220)
DT=0.05
F=100.
X(1)=0.0
Y(1)=0.0
DY(1)=0.0
DX(1)=0.
T(1)=0.0
I=2
Z(1)=0.0
5  T(I)=T(I-1)+DT
   IF(T(I).GT.4.) GO TO 10
   Z(I)=10.*SIN(3.14*T(I))
   GO TO 15
10  Z(I)=0.0
15  IF(I.GT.2) GOTO 20
   X(I)=(400.*6.*DT**4*Z(I))/(36.+840.*DT**2+2400.*DT**4)
   Y(I)=(1+40./6.*DT**2)*X(I)/(40./6.*DT**2)
   GO TO 50
20  X(I)=DX(I-1)*DT**2+2.*X(I-1)-X(I-2)
   Y(I)=DY(I-1)*DT**2+2.*Y(I-1)-Y(I-2)
50  DX(I)=-40.*X(I)+40.*Y(I)
   DY(I)=40.*X(I)-100.*Y(I)+60.*Z(I)
   I=I+1
   IF(I.LE.202) GOTO 5
   DO 70 I=1,202
   PRINT60,T(I),Z(I),X(I),Y(I)
60  FORMAT(10X,4F12.4)
70  CONTINUE
   CALL EZPLOT(T,X,202)
   CALL EZPLOT(T,Y,-202)
   CALL FINISH
   STOP
   END
```

Prob. 5-50 Cont.



5-51

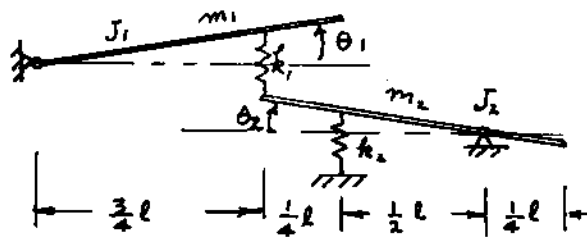
$$\left. \begin{aligned} J_1 \ddot{\theta}_1 &= K_1(\theta_2 - \theta_1) \\ J_2 \ddot{\theta}_2 &= -K_1(\theta_2 - \theta_1) + K_2(\theta_3 - \theta_2) \\ J_3 \ddot{\theta}_3 &= -K_2(\theta_3 - \theta_2) \end{aligned} \right\} \begin{aligned} &\text{If } \phi = (\theta_2 - \theta_1) \\ &\psi = (\theta_3 - \theta_2) \\ &\text{the DEs. can be} \\ &\text{written as} \end{aligned}$$

$$\left. \begin{aligned} J_2 \ddot{\phi} &= -K_1 \left(1 + \frac{J_2}{J_1}\right) \phi + K_2 \psi \\ J_3 \ddot{\psi} &= K_1 \frac{J_3}{J_2} \phi - K_2 \left(1 + \frac{J_3}{J_2}\right) \psi \end{aligned} \right\} \begin{aligned} &\text{DEs in two} \\ &\text{coordinates} \end{aligned}$$

Freq. Eq. from either set of equations is

$$\omega^4 - \left[\frac{K_1}{J_1} + \frac{K_2}{J_2} \left(1 + \frac{K_1}{K_2} + \frac{J_2}{J_3}\right) \right] \omega^2 + \frac{K_1 K_2}{J_1 J_2} \left(\frac{J_1 + J_2 + J_3}{J_3} \right) = 0$$

5-52



$$J_1 \ddot{\theta}_1 = -\frac{3}{4} l k_1 (\theta_1 - \theta_2) \frac{3}{4} l$$

$$J_2 \ddot{\theta}_2 = \frac{3}{4} l k_1 (\theta_1 - \theta_2) \frac{3}{4} l - \frac{1}{2} l k_2 \theta_2 \frac{1}{2} l$$

$$\begin{bmatrix} J_1 & 0 \\ 0 & J_2 \end{bmatrix} \begin{Bmatrix} \ddot{\theta}_1 \\ \ddot{\theta}_2 \end{Bmatrix} + l^2 k_1 \begin{bmatrix} \frac{9}{16} & -\frac{9}{16} \\ -\frac{9}{16} & (\frac{9}{16} + \frac{1}{4} \frac{k_2}{k_1}) \end{bmatrix} \begin{Bmatrix} \theta_1 \\ \theta_2 \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \end{Bmatrix}$$

$$J_1 = m_1 l^2 \frac{1}{3}$$

$$J_2 = m_2 l^2 \left(\frac{1}{12} + \frac{1}{16} \right)$$

5-53

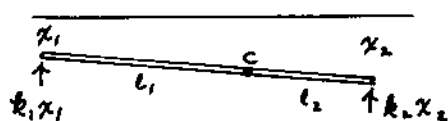
$$\begin{bmatrix} \left[K_1 \left(1 + \frac{J_2}{J_1}\right) - \omega^2 J_2 \right] & -K_2 \\ -K_1 \frac{J_3}{J_2} & \left[K_2 \left(1 + \frac{J_3}{J_2}\right) - \omega^2 J_3 \right] \end{bmatrix} \begin{Bmatrix} \phi \\ \psi \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \end{Bmatrix}$$

$$\frac{\phi}{\psi} = \frac{K_2}{K_1 \left(1 + \frac{J_2}{J_1}\right) - \omega^2 J_2} = \frac{\theta_2 - \theta_1}{\theta_3 - \theta_2} = \frac{\frac{\theta_2}{\theta_1} - 1}{\frac{\theta_3}{\theta_1} - \frac{\theta_2}{\theta_1}}$$

Assign numerical values for K_1, K_2, J_1, J_2, J_3 and prove

$$J_1 \theta_1^{(i)} \theta_1^{(j)} + J_2 \theta_2^{(i)} \theta_2^{(j)} + J_3 \theta_3^{(i)} \theta_3^{(j)} = 0$$

5-54



$$\begin{aligned} \text{Displ. of } C &= x_1 + \frac{l_1}{l_1 + l_2} (x_2 - x_1) \\ &= \frac{l_2 x_1 + l_1 x_2}{l_1 + l_2} \end{aligned}$$

$$T = \frac{1}{2} m \left(\frac{l_2 \dot{x}_1 + l_1 \dot{x}_2}{l_1 + l_2} \right)^2 + \frac{1}{2} J_C \left(\frac{\dot{x}_2 - \dot{x}_1}{l_1 + l_2} \right)^2$$

$$U = \frac{1}{2} k_1 x_1^2 + \frac{1}{2} k_2 x_2^2 \quad \therefore \text{Dynamic coupling exists}$$

5-55

Equations to be solved are:

$$\ddot{x} = -720x + 360y \quad (\text{see p132})$$

$$\ddot{y} = 1440(x - y) + 160$$

$$(s^2 + 720)\bar{x}(s) - 360\bar{y}(s) = 0$$

$$-1440\bar{x}(s) + (s^2 + 1440)\bar{y}(s) = \frac{160}{s}$$

$$\text{from 1st eq. } \bar{x}(s) = \left(\frac{360}{s^2 + 720} \right) \bar{y}(s)$$

$$\text{subst. into 2nd eq. } -1440 \left(\frac{360}{s^2 + 720} \right) \bar{y}(s) + (s^2 + 1440)\bar{y}(s) = \frac{160}{s}$$

x by $(s^2 + 720)$

$$[-518,400 + s^4 + 2160s^2 + 1,036,800] \bar{y}(s) = \frac{160(s^2 + 720)}{s}$$

$$[s^4 + 2160s^2 + 518,400] \bar{y}(s) = \frac{160}{s} (s^2 + 720)$$

$$\bar{y}(s) = \frac{160(s^2 + 720)}{s(s^4 + 2160s^2 + 518,400)}$$

$$\text{Roots: } s^2 = -1080 \pm \sqrt{1,166,400 - 518,400}$$

$$= -1080 \pm \sqrt{648,000}$$

$$= -1080 \pm 804.9845$$

also $s = 0$ is a root.

5-55 Cont.

$$\therefore s_1^2 = -275.0155 \quad s_1 = \pm i 16.5836$$

$$s_2^2 = -1884.9845 \quad s_2 = \pm i 43.4164$$

Rewrite $\bar{y}(s)$ as

$$\begin{aligned}\bar{y}(s) &= \frac{160(s^2 + 720)}{s(s^2 + 275.0155)(s^2 + 1884.9845)} \\&= \frac{160(s^2 + 720)}{s(s + i 16.5836)(s - i 16.5836)(s + i 43.4164)(s - i 43.4164)} \\&= \frac{C_1}{(s + i 16.5836)} + \frac{C_2}{(s - i 16.5836)} + \frac{C_3}{(s + i 43.4164)} \\&\quad + \frac{C_4}{(s - i 43.4164)} + \frac{C_5}{s}\end{aligned}$$

$$\begin{aligned}C_1 &= \lim_{s \rightarrow s_1} \left[\frac{160(s^2 + 720)(s - i 16.5836)}{s(s + i 16.5836)(s - i 16.5836)(s^2 + 1884.9845)} \right] \\&= \frac{160(-275.0155 + 720)}{(-i 16.5836)(-i 33.1672)(-275.0155 + 1884.9845)} = -.0804007\end{aligned}$$

$$C_2 = \lim_{s \rightarrow s_1} \left[\frac{160(s^2 + 720)(s + i 16.5836)}{s(s + i 16.5836)(s - i 16.5836)(s^2 + 1884.98)} \right] = -.0804007$$

$$C_3 = \lim_{s \rightarrow s_2} \left[\frac{160(s^2 + 720)(s + i 43.4164)}{s(s^2 + 275.0155)(s + i 43.4164)(s - i 43.4164)} \right] = -.03071$$

$$C_4 = C_3$$

$$C_5 = \frac{160(720)}{(275.0155)(1884.9845)} = 0.2222$$

5-55 Cont.

$$\bar{y}(s) = \frac{.2222}{s} - \frac{.0804007}{(s+i16.5836)} - \frac{.0804007}{(s-i16.5836)} \\ - \frac{.03071}{(s+i43.4164)} - \frac{.03071}{(s-i43.4164)}$$

Inverse

$$y(t) = 0.2222 - .0804007 \left(e^{-i16.5836t} + e^{i16.5836t} \right) \\ - .03071 \left(e^{-i43.4164t} + e^{i43.4164t} \right)$$

$$= 0.2222 - .1608 \cos 16.58t - .06142 \cos 43.41t$$

$$= .1608 (1 - \cos 16.58t) + .06142 (1 - \cos 43.41t) \text{ meters}$$

move dec. pt. 2 places to left for centimeters

Solution for $x(t)$

$$\bar{x}(s) = \frac{360}{(s^2 + 720)} \quad \bar{y}(s) = \frac{360}{(s^2 + 720)} \cdot \frac{160(s^2 + 720)}{5(s^2 + 275.0155)(s^2 + 1884.9845)}$$

\therefore roots are same. Solve as above with partial fractions, the equation

$$\bar{x}(s) = \frac{57600}{5(s^2 + 275.0155)(s^2 + 1884.9845)}$$

Ans.

$$x(t) = .1301(1 - \cos 16.58t) - .0190(1 - \cos 43.41t) \text{ meters.}$$

5-56

Examine for example the subsidiary sol. for Prob 5-55

$$\bar{y}(s) = \sum \frac{C_i}{s - s_i} \quad \text{where } s_i \text{ are roots}$$

\therefore Results are sums of normal modes

$$m \begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix} \begin{Bmatrix} \ddot{x}_1 \\ \ddot{x}_2 \end{Bmatrix} + k \begin{bmatrix} 2 & -1 \\ -1 & 2 \end{bmatrix} \begin{Bmatrix} x_1 \\ x_2 \end{Bmatrix} = \begin{Bmatrix} F_0 \\ 0 \end{Bmatrix} \sin \omega t$$

$$ms^2 \begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix} \begin{Bmatrix} \bar{x}_1(s) \\ \bar{x}_2(s) \end{Bmatrix} + k \begin{bmatrix} 2 & -1 \\ -1 & 2 \end{bmatrix} \begin{Bmatrix} \bar{x}_1(s) \\ \bar{x}_2(s) \end{Bmatrix} = \begin{Bmatrix} \frac{F_0 \omega}{s^2 + \omega^2} + msx_1(0) + m\dot{x}_1(0) \\ msx_2(0) + m\dot{x}_2(0) \end{Bmatrix}$$

$$\begin{bmatrix} (ms^2 + 2k) & -k \\ -k & (ms^2 + 2k) \end{bmatrix} \begin{Bmatrix} \bar{x}_1(s) \\ \bar{x}_2(s) \end{Bmatrix} = \begin{Bmatrix} \text{ } \\ \text{ } \end{Bmatrix}$$

Det. of matrix $(ms^2 + 2k)^2 - k^2 = 0$, $s^2 = -\frac{2k}{m} \pm \frac{k}{m}$

$$\therefore (s^2 + \frac{k}{m})(s^2 + \frac{3k}{m})$$

$$\bar{x}_1(s) = \frac{\left[\frac{F_0 \omega}{s^2 + \omega^2} + msx_1(0) + m\dot{x}_1(0) \right] - k}{(s^2 + \frac{k}{m})(s^2 + \frac{3k}{m})m}$$

$$\bar{x}_2(s) = \frac{\begin{vmatrix} (ms^2 + 2k) & \left[\frac{F_0 \omega}{s^2 + \omega^2} + msx_1(0) + m\dot{x}_1(0) \right] \\ -k & [msx_2(0) + m\dot{x}_2(0)] \end{vmatrix}}{(s^2 + \frac{k}{m})(s^2 + \frac{3k}{m})m}$$

For steady state vibration $x(0)$, $\dot{x}(0) = 0$ and $s = i\omega$

$$x_1(t) = \frac{(2\frac{k}{m} - \omega^2) F_0 \sin \omega t}{(\frac{k}{m} - \omega^2)(3\frac{k}{m} - \omega^2)}$$

$$x_2(t) = \frac{k F_0 \sin \omega t}{(\frac{k}{m} - \omega^2)(3\frac{k}{m} - \omega^2) m}$$

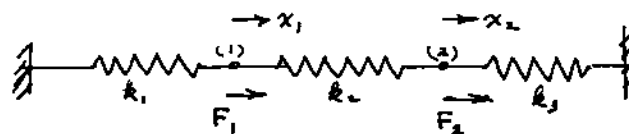
6-1

$$F_1 = -k_1 x_1 - k_2 (x_1 - x_2)$$

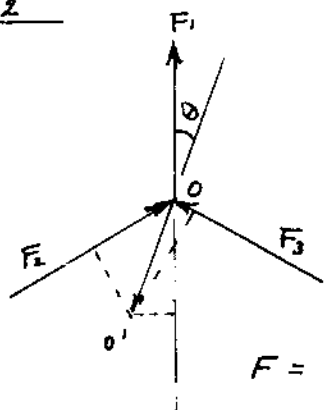
$$F_2 = -k_3 x_2 - k_2 (x_2 - x_1)$$

$$\begin{Bmatrix} F_1 \\ F_2 \end{Bmatrix} = \begin{bmatrix} -(k_1+k_2) & k_2 \\ k_2 & -(k_2+k_3) \end{bmatrix} \begin{Bmatrix} x_1 \\ x_2 \end{Bmatrix}$$

$$\begin{Bmatrix} x_1 \\ x_2 \end{Bmatrix} = [K]^{-1} \begin{Bmatrix} F_1 \\ F_2 \end{Bmatrix} = \frac{-1}{k_1 k_2 + k_1 k_3 + k_2 k_3} \begin{bmatrix} (k_2+k_3) & k_2 \\ k_2 & (k_1+k_2) \end{bmatrix} \begin{Bmatrix} F_1 \\ F_2 \end{Bmatrix}$$



6-2

Let O move to O' with δ small

$$\text{Then } F_1 = \delta k \cos \theta$$

$$F_2 = \delta k \cos (60 - \theta)$$

$$F_3 = \delta k \cos (60 + \theta)$$

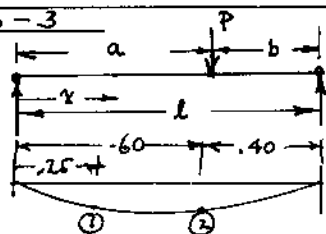
Force along δ is

$$F = F_1 \cos \theta + F_2 \cos (60 - \theta) + F_3 \cos (60 + \theta)$$

$$= k \delta [\cos^2 \theta + \cos^2 (60 - \theta) + \cos^2 (60 + \theta)] = 1.5 k \delta$$

$$\therefore \frac{\delta}{F} = \frac{1}{1.5k} \text{ and independent of } \theta$$

6-3



$$y = \frac{Pbx}{6EI} (l^2 - x^2 - b^2) \quad x \geq a$$

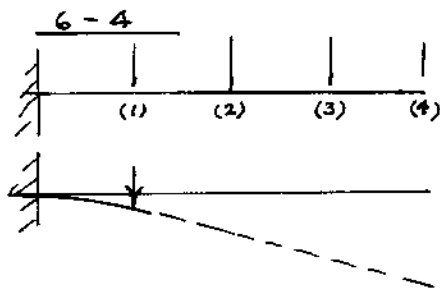
$$\text{Position ①} = 0.25l, \quad \text{position ②} = 0.60l$$

$$a_{11} = \frac{(0.75)(0.25)l^3}{6EI} (1 - 0.25^2 - 0.75^2) = 0.0114 \frac{l^3}{EI}$$

$$a_{12} = a_{21} = \frac{(0.15)(0.40)l^3}{6EI} (1 - 0.40^2 - 0.25^2) = 0.0130 \frac{l^3}{EI}$$

$$a_{22} = \frac{(0.40)(0.60)l^3}{6EI} (1 - 0.60^2 - 0.40^2) = 0.0192 \frac{l^3}{EI}$$

$$a = \frac{l^3}{EI} \begin{bmatrix} 0.0114 & 0.0130 \\ 0.0130 & 0.0192 \end{bmatrix}$$



$$a_{11} = \frac{\left(\frac{l}{4}\right)^3}{3EI} = \frac{l^3}{192EI}$$

$$\theta_{11} = \frac{\left(\frac{l}{4}\right)^2}{2EI} = \frac{l^2}{32EI}$$

$$a_{21} = a_{11} + \theta_{11} \frac{l}{4} = \frac{l^3}{192EI} + \frac{l^3}{128EI} = \frac{2.50 l^3}{192EI}$$

$$a_{31} = a_{21} + \frac{l^3}{128EI} = \frac{4 l^3}{192EI}$$

$$a_{41} = a_{31} + \frac{l^3}{128EI} = \frac{5.50 l^3}{192EI}$$

$$a_{22} = \frac{\left(\frac{l}{2}\right)^3}{3EI} = \frac{l^3}{24EI}$$

$$\theta_{22} = \frac{\left(\frac{l}{2}\right)^2}{2EI} = \frac{l^2}{8EI}$$

$$a_{32} = a_{22} + \theta_{22} \frac{l}{4} = \left(\frac{1}{24} + \frac{1}{32}\right) \frac{l^3}{EI} = \frac{7}{96} \frac{l^3}{EI}$$

$$a_{42} = \left(\frac{7}{96} + \frac{1}{32}\right) \frac{l^3}{EI} = \frac{10}{96} \frac{l^3}{EI}$$

$$a_{33} = \frac{\left(\frac{3l}{4}\right)^3}{3EI} = \frac{9}{64} \frac{l^3}{EI}$$

$$\theta_{33} = \frac{\left(\frac{3l}{4}\right)^2}{2EI} = \frac{9}{32} \frac{l^2}{EI}$$

$$a_{43} = a_{33} + \theta_{33} \frac{l}{4} = \left(\frac{9}{64} + \frac{9}{128}\right) \frac{l^3}{EI} = \frac{27}{128} \frac{l^3}{EI}$$

$$a_{44} = \frac{l^3}{3EI}$$

$$a = \frac{l^3}{EI} \begin{bmatrix} \frac{1}{192} & \frac{2.5}{192} & \frac{4}{192} & \frac{5.5}{192} \\ \frac{2.5}{192} & \frac{1}{24} & \frac{7}{96} & \frac{10}{96} \\ \frac{4.0}{192} & \frac{7}{96} & \frac{9}{64} & \frac{27}{128} \\ \frac{5.5}{192} & \frac{10}{96} & \frac{27}{128} & \frac{1}{3} \end{bmatrix} = \frac{l^3}{192EI} \begin{bmatrix} 1 & 2.5 & 4.0 & 5.5 \\ 2.5 & 8.0 & 14.0 & 20.0 \\ 4.0 & 14.0 & 27.0 & 40.5 \\ 5.5 & 20.0 & 40.5 & 64.0 \end{bmatrix}$$

6-4 Cont:

Computer Program for Matrix Inversion

```

C      PROBLEM 6-4 THOMSON
1      REAL A(4,4), AINV(4,4), WKAREA(4)
2      DO 10 I=1,4
3          READ, (A(I,J), J=1,4)
4      10 CONTINUE
5          DO 16 J=1,4
6              PRINT15, (A(I,J), J=1,4)
7      15 FORMAT(10X,4F12.4)
8      16 CONTINUE
9      N=4
10     IA=4
11     IDGT=0
12     CALL LINVIF (A,N,IA,AINV,IDGT,WKAREA,IER)
13     DO 26 I=1,4
14         PRINT25, (AINV(I,J), J=1,4)
15     25 FORMAT(' ',10X,4F12.4)
16     26 CONTINUE
17     STOP
18     END
```

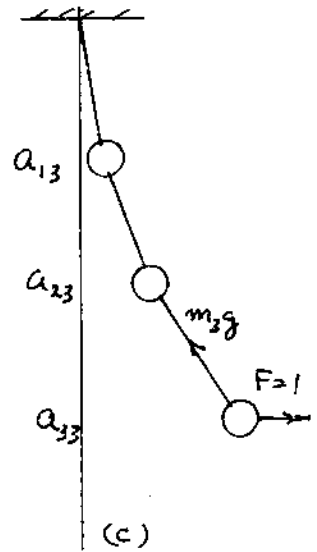
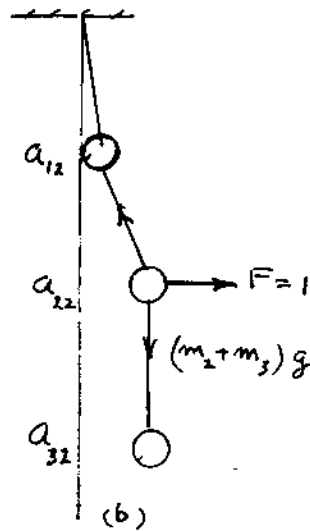
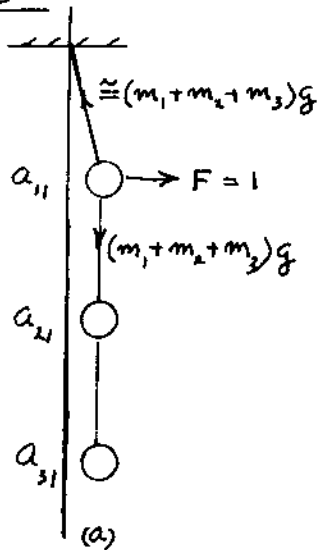
Original Matrix

1.0000	2.5000	4.0000	5.5000
2.5000	8.0000	14.0000	20.0000
4.0000	14.0000	27.0000	40.5000
5.5000	20.0000	40.5000	64.0000

Inverted Matrix

6.2680	-3.9381	1.4845	-0.2474
-3.9381	4.7835	-3.1959	0.8660
1.4845	-3.1959	3.2990	-1.2165
-0.2474	0.8660	-1.2165	0.5361

6-5



From (a)

$$\frac{a_{11}}{l_1} (m_1 + m_2 + m_3)g = 1$$

$$a_{11} = a_{21} = a_{31} = \frac{1}{m_1 + m_2 + m_3} \cdot \frac{l_1}{g}$$

From (b)

$$\left(\frac{a_{22} - a_{12}}{l_2} \right) (m_2 + m_3)g = 1$$

$$a_{22} = \frac{1}{m_2 + m_3} \cdot \frac{l_2}{g} + \frac{1}{m_1 + m_2 + m_3} \cdot \frac{l_1}{g} = a_{32}$$

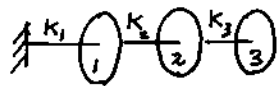
From (c)

$$\left(\frac{a_{33} - a_{23}}{l_3} \right) m_3 g = 1$$

$$a_{33} = \frac{1}{m_3} \cdot \frac{l_3}{g} + \frac{1}{m_2 + m_3} \cdot \frac{l_2}{g} + \frac{1}{m_1 + m_2 + m_3} \cdot \frac{l_1}{g}$$

$$a_{ij} = a_{ji}$$

6-6



Give each disk a unit rotation holding other disks with zero rotation. Torque required is then:

$$\theta_1 = 1.0 \quad T_1 = (K_1 + K_2), \quad T_2 = -K_1, \quad T_3 = 0$$

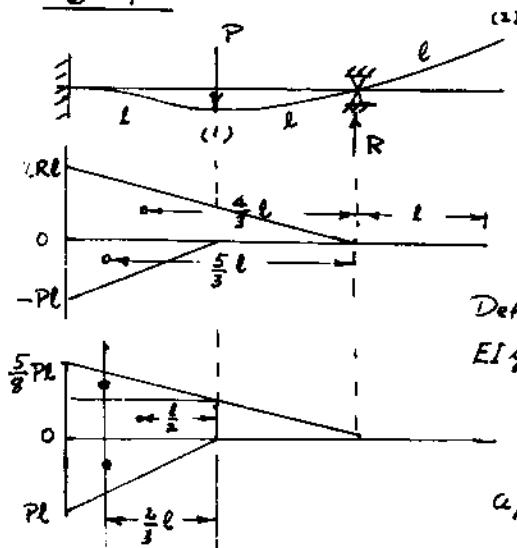
$$\theta_2 = 1.0 \quad T_1 = -K_2, \quad T_2 = (K_2 + K_3), \quad T_3 = -K_3$$

$$\theta_3 = 1.0 \quad T_1 = 0, \quad T_2 = -K_3, \quad T_3 = K_3$$

$$[K] = \begin{bmatrix} (K_1 + K_2) & -K_2 & 0 \\ -K_2 & (K_2 + K_3) & -K_3 \\ 0 & -K_3 & K_3 \end{bmatrix} \quad [a] = \begin{bmatrix} \frac{1}{K_1} & \frac{1}{K_1} & \frac{1}{K_1} \\ \frac{1}{K_1} & (\frac{1}{K_1} + \frac{1}{K_2}) & (\frac{1}{K_1} + \frac{1}{K_2}) \\ \frac{1}{K_1} & (\frac{1}{K_1} + \frac{1}{K_2}) & (\frac{1}{K_1} + \frac{1}{K_2} + \frac{1}{K_3}) \end{bmatrix}$$

For flexibility apply unit torque to each disk in turn and measure rotation.

6-7



Defl. at R = 0

$$\frac{1}{2} (2Rl \times 2l) \frac{4}{3} l - \frac{1}{2} (Pl \times l) \frac{5}{3} l = 0$$

$$\therefore R = \frac{5}{16} P$$

$$2Rl = \frac{5}{8} Pl$$

Defl. at (1) = moment of area about (1)

$$EI \theta_1 = \left(\frac{5}{16} Pl \times l \right) \frac{l}{2} + \frac{1}{2} \left(\frac{5}{16} Pl^2 \right) \frac{2}{3} l - \frac{1}{2} (Pl^2) \frac{2}{3} l$$

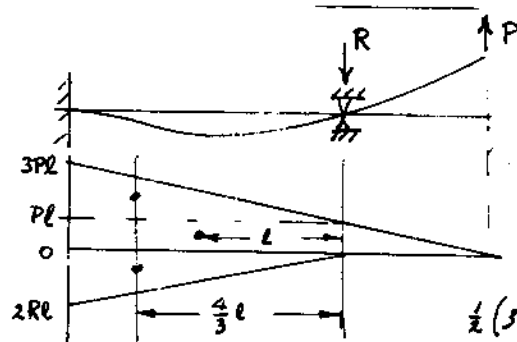
$$= Pl^3 \left(\frac{5}{32} + \frac{5}{48} - \frac{1}{3} \right) = - \frac{3.5}{48} Pl^3$$

$$a_{11} = \frac{3.50}{48} \frac{l^3}{EI} = \frac{7}{96} \frac{l^3}{EI}$$

Defl. at (2) = moment of area about (2)

$$EI \theta_2 = \frac{1}{2} \left[\frac{5}{8} Pl \times 2l \right] \frac{7}{3} l - \frac{1}{2} [Pl^2] \frac{8}{3} l = Pl^3 \left(\frac{35}{24} - \frac{8}{6} \right) = \frac{1}{8} Pl^3$$

$$\therefore a_{21} = a_{12} = \frac{1}{8} \frac{l^3}{EI}$$



Defl at R = 0

$$(2Pl^2) l + \frac{1}{2} (2Pl \times 2l) \frac{4}{3} l - \frac{1}{2} (2Rl \times 2l) \frac{4}{3} l = 0$$

$$Pl^3 \left(2 + \frac{8}{3} \right) = Rl^3 \left(\frac{8}{3} \right) \therefore R = \frac{7}{4} P$$

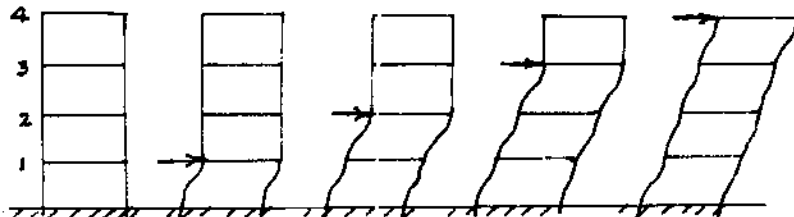
For a_{22}

$$\frac{1}{2} (3Pl \times 3l) 2l - \frac{1}{2} \left(\frac{7}{2} Pl \times 2l \right) \left(l + \frac{4}{3} l \right) = EI \theta_2$$

$$Pl^3 (9 - 8.666) = \left(\frac{5.7}{6} - \frac{49}{6} \right) Pl^3 = \frac{5}{6} Pl^3 \therefore a_{22} = \frac{5}{6} \frac{l^3}{EI}$$

$$[a] = \frac{l^3}{EI} \begin{bmatrix} 7/96 & 1/8 \\ 1/8 & 5/6 \end{bmatrix}$$

6-8



$$a_{11} = \frac{1}{12} \frac{l^3}{EI}$$

$$a_{12} = \frac{1}{12} \frac{l^3}{EI}$$

$$a_{13} = \frac{1}{12} \frac{l^3}{EI}$$

$$a_{14} = \frac{1}{12} \frac{l^3}{EI}$$

$$a_{21} = \frac{1}{12} "$$

$$a_{22} = \frac{3}{12} "$$

$$a_{23} = \frac{2}{12} "$$

$$a_{24} = \frac{2}{12} "$$

$$a_{31} = \frac{1}{12} "$$

$$a_{32} = \frac{2}{12} "$$

$$a_{33} = \frac{3}{12} "$$

$$a_{34} = \frac{2}{12} "$$

$$a_{41} = \frac{1}{12} "$$

$$a_{42} = \frac{2}{12} "$$

$$a_{43} = \frac{2}{12} "$$

$$a_{44} = \frac{4}{12} "$$

$$[a] = \frac{l^3}{12EI} \begin{bmatrix} 1 & 1 & 1 & 1 \\ 1 & 2 & 2 & 2 \\ 1 & 2 & 3 & 3 \\ 1 & 2 & 3 & 4 \end{bmatrix}$$

6-8 Cont.

$$[k] = [a]^{-1}$$

$$D = \det. [a] = \begin{vmatrix} 2 & 2 & 2 \\ 2 & 3 & 3 \\ 2 & 3 & 4 \end{vmatrix} - \begin{vmatrix} 1 & 2 & 2 \\ 1 & 3 & 3 \\ 1 & 3 & 4 \end{vmatrix} + \begin{vmatrix} 1 & 2 & 2 \\ 1 & 2 & 3 \\ 1 & 2 & 4 \end{vmatrix} - \begin{vmatrix} 1 & 2 & 2 \\ 1 & 2 & 3 \\ 1 & 2 & 3 \end{vmatrix}$$

$$= 2(12-9) - 1(12-9) + 1(8-6) - 1(0) - 2(8-6) + 2(4-3) - 2(4-3) + 0 + 0 - 0 + 0 - 0 = 1$$

$$C_{11} = \begin{vmatrix} 2 & 2 & 2 \\ 2 & 3 & 3 \\ 2 & 3 & 4 \end{vmatrix} \quad C_{12} = -\begin{vmatrix} 1 & 2 & 2 \\ 1 & 3 & 3 \\ 1 & 3 & 4 \end{vmatrix} \quad C_{13} = \begin{vmatrix} 1 & 2 & 2 \\ 1 & 2 & 3 \\ 1 & 2 & 4 \end{vmatrix} = 0 \quad C_{14} = 0$$

$$C_{21} = -\begin{vmatrix} 1 & 1 & 1 \\ 2 & 3 & 3 \\ 2 & 3 & 4 \end{vmatrix} \quad C_{22} = \begin{vmatrix} 1 & 1 & 1 \\ 1 & 3 & 3 \\ 1 & 3 & 4 \end{vmatrix} \quad C_{23} = -\begin{vmatrix} 1 & 1 & 1 \\ 1 & 2 & 3 \\ 1 & 2 & 4 \end{vmatrix} \quad C_{24} = 0$$

$$C_{31} = 0 \quad C_{32} = -\begin{vmatrix} 1 & 1 & 1 \\ 1 & 2 & 2 \\ 1 & 3 & 4 \end{vmatrix} \quad C_{33} = \begin{vmatrix} 1 & 1 & 1 \\ 1 & 2 & 2 \\ 1 & 2 & 4 \end{vmatrix} \quad C_{34} = -\begin{vmatrix} 1 & 1 & 1 \\ 1 & 2 & 2 \\ 1 & 2 & 3 \end{vmatrix}$$

$$C_{41} = 0 \quad C_{42} = 0 \quad C_{43} = -\begin{vmatrix} 1 & 1 & 1 \\ 1 & 2 & 2 \\ 1 & 2 & 3 \end{vmatrix} \quad C_{44} = \begin{vmatrix} 1 & 1 & 1 \\ 1 & 2 & 2 \\ 1 & 2 & 3 \end{vmatrix}$$

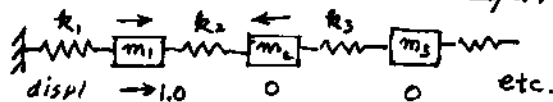
$$C'_{11} = C_{11} = 2, \quad C'_{12} = -1 \quad C'_{21} = -1, \quad C'_{22} = 2 \quad C'_{23} = -1$$

$$C'_{32} = -1 \quad C'_{33} = 2 \quad C'_{34} = -1 \quad C'_{43} = -1 \quad C'_{44} = 1$$

$$\therefore [k] = \frac{12EI}{l^3} \begin{bmatrix} 2 & -1 & 0 & 0 \\ -1 & 2 & -1 & 0 \\ 0 & -1 & 2 & -1 \\ 0 & 0 & -1 & 1 \end{bmatrix}$$

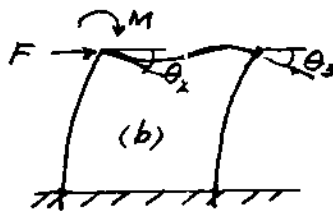
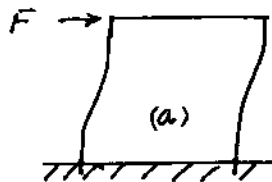
6-9

Give each mass a unit displ. keeping all others at zero. Then measure force required. ie for $x_1 = 1.0$



$$[k] = \begin{bmatrix} (k_1 + k_2) & -k_2 & 0 & 0 & 0 & \dots \\ -k_2 & (k_2 + k_3) & -k_3 & 0 & 0 & \dots \\ 0 & -k_3 & (k_3 + k_4) & -k_4 & 0 & \dots \\ 0 & 0 & -k_4 & (k_4 + k_5) & -k_5 & \dots \\ \vdots & \vdots & \vdots & \vdots & \vdots & \ddots \end{bmatrix}$$

6-10



$$(a) \quad k = 2 \times \frac{12EI}{l^3} = 24 \frac{EI}{l^3}$$

$$(b) \quad F = \frac{EI}{l^3} [24 - 6\theta_1 l - 6\theta_2 l] \quad \theta_1 = \theta_2 = \theta$$

$$M = \frac{EI}{l^2} [-6 + 8\theta_1 l + 2\theta_2 l] = 0 \quad (\text{see Prob 6-11})$$

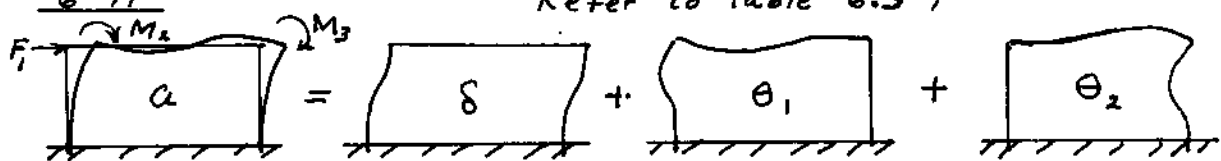
$$-6 + 10\theta l = 0 \quad \therefore \theta = \frac{6}{10l}$$

$$F = \frac{EI}{l^3} [24 - 12 \times \frac{6}{10}] = 16.8 \frac{EI}{l^3}$$

$$\text{Ratio } \frac{(a)}{(b)} = \frac{24}{16.8}$$

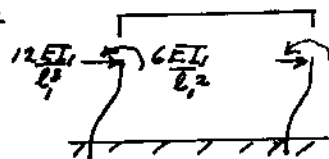
6-11

Refer to Table 6.3-1



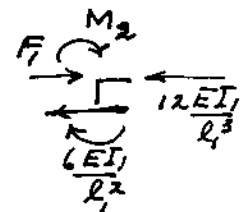
a is superposition of $\delta + \theta_1 + \theta_2$

δ mode



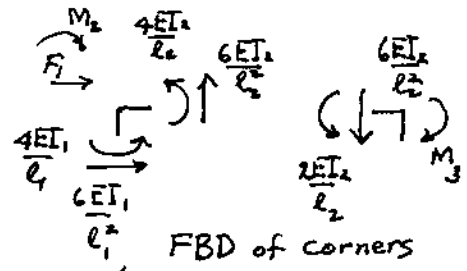
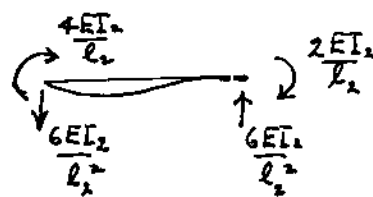
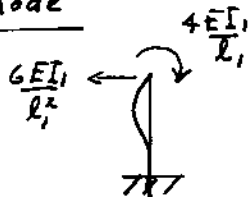
$$\therefore F_1 = 24 \frac{EI_1}{l_1^3} \delta$$

FBD of corner



$$M_2 = M_3 = -6 \frac{EI_1}{l_1^2} \delta$$

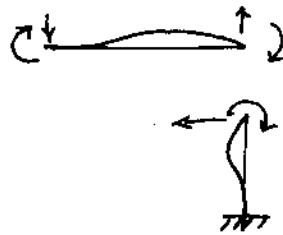
θ_1 mode



$$F_1 = -\frac{6EI_1}{l_1^2} \theta_1, \quad M_2 = \left(\frac{4EI_1}{l_1} + \frac{4EI_2}{l_2} \right) \theta_1, \quad M_3 = \left(\frac{2EI_2}{l_2} \right) \theta_1$$

6-11 Cont

θ_2 mode

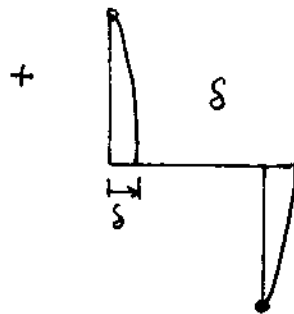
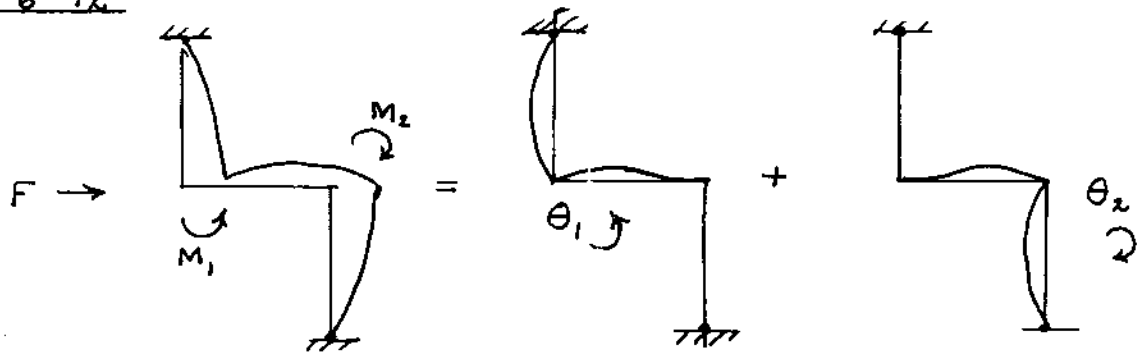


Same as mode θ_1 ,
but shifted to right corner

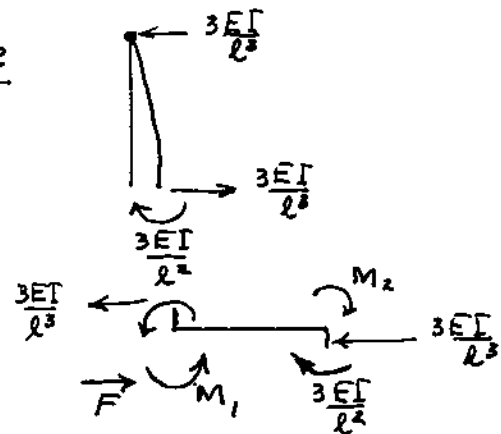
Sum results for δ , θ_1 , & θ_2

$$\begin{Bmatrix} F_1 \\ M_1 \\ M_2 \end{Bmatrix} = \begin{bmatrix} \frac{24EI_1}{l_1^3} & -\frac{6EI_1}{l_1^2} & -\frac{6EI_1}{l_1^2} \\ -\frac{6EI_1}{l_1^2} & (\frac{4EI_1}{l_1} + \frac{4EI_2}{l_2}) & \frac{2EI_2}{l_2} \\ -\frac{6EI_1}{l_1^2} & \frac{2EI_2}{l_2} & (\frac{4EI_1}{l_1} + \frac{4EI_2}{l_2}) \end{bmatrix} \begin{Bmatrix} \delta \\ \theta_1 \\ \theta_2 \end{Bmatrix}$$

6-12



δ -mode



$$F = \frac{6EI}{l^3} \delta$$

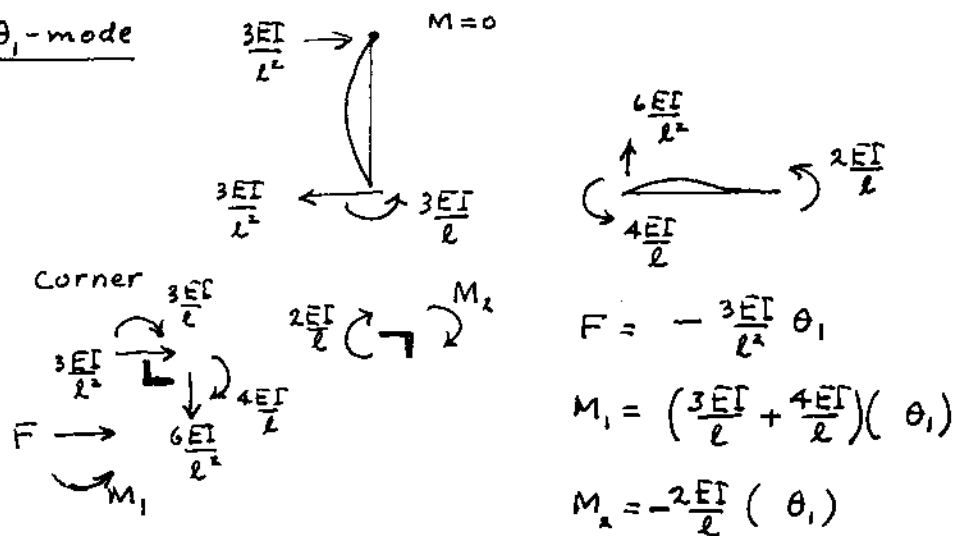
$$M_1 = -\frac{3EI}{l^2} \delta$$

$$M_2 = -\frac{3EI}{l^2} \delta$$

from corners

6-12 Cont

θ_1 -mode



$$F = -\frac{3EI}{l^2} \theta_1$$

$$M_1 = \left(\frac{3EI}{l} + \frac{4EI}{l} \right) (\theta_1)$$

$$M_2 = -\frac{2EI}{l} (\theta_1)$$

θ_2 -mode

this mode is same as above moved to right corner with θ_1 replaced by $+\theta_2$

Add results

$$\begin{Bmatrix} F \\ M_1 \\ M_2 \end{Bmatrix} = \begin{bmatrix} \frac{6EI}{l^3} & -\frac{3EI}{l^2} & -\frac{3EI}{l^2} \\ -\frac{3EI}{l^2} & \left(\frac{3EI}{l} + \frac{4EI}{l} \right) & -\frac{2EI}{l} \\ -\frac{3EI}{l^2} & -\frac{2EI}{l} & \left(\frac{3EI}{l} + \frac{4EI}{l} \right) \end{bmatrix} \begin{Bmatrix} \delta \\ \theta_1 \\ \theta_2 \end{Bmatrix}$$

Since no moments were applied at corners $M_1 = M_2 = 0$

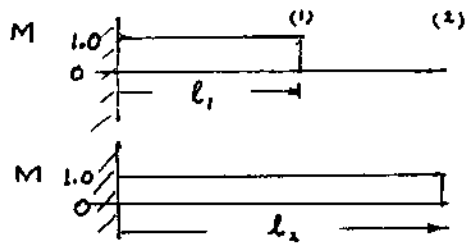
$$\left. \begin{aligned} 0 &= -\frac{3EI}{l^2} \delta + \frac{7EI}{l} \theta_1 - \frac{2EI}{l} \theta_2 \\ 0 &= -\frac{3EI}{l^2} \delta - \frac{2EI}{l} \theta_1 + \frac{7EI}{l} \theta_2 \end{aligned} \right\} \begin{array}{l} \text{Subtr. we find} \\ \theta_1 = \theta_2 = \theta \\ \text{as expected by} \\ \text{Symmetry} \end{array}$$

$$M_1 = -\frac{3EI}{l^2} \delta + \frac{5EI}{l} \theta = 0 \quad \therefore \theta = \frac{3\delta}{5l}$$

$$F = \frac{6EI}{l^3} \delta - \frac{6EI}{l^2} \left(\frac{3\delta}{5l} \right) = \underline{\underline{2.40 \frac{EI}{l^3} \delta}}$$

6-13

Moment Diagram



$$EI \theta_{21} = EI \theta_{12} = M_1 l_1 = M_2 l_2$$

$$M_1 = M_2 = 1$$

$$\therefore \theta_{12} = \theta_{21}$$

By Work done

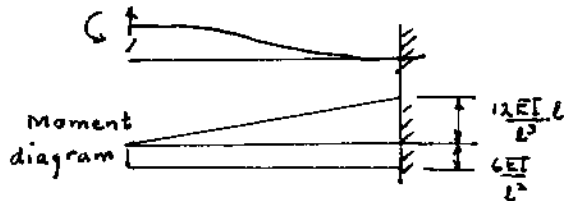
 M_1 1st followed by M_2

$$W = \frac{1}{2} M_1 \theta_{11} + \frac{1}{2} M_2 \theta_{22} + M_1 \theta_{12}$$

 M_2 1st " " M_1

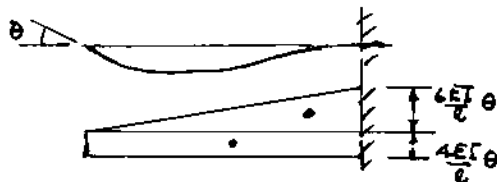
$$W = \frac{1}{2} M_2 \theta_{22} + \frac{1}{2} M_1 \theta_{11} + M_2 \theta_{21}$$

6-14



Difference in slope = 0

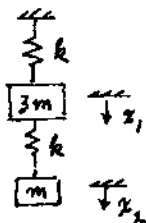
$$\left(\frac{1}{2} \cdot \frac{12EI}{l^3} \cdot l \right) - \left(\frac{6EI}{l^2} \cdot l \right) = 0$$



Difference in defl. = 0

$$\left(\frac{1}{2} \cdot \frac{6EI}{l} \right) \frac{2}{3} l - \left(\frac{4EI}{l} \cdot l \right) \frac{l}{2} = 0$$

6-15



$$m \begin{bmatrix} 3 & 0 \\ 0 & 1 \end{bmatrix} \begin{Bmatrix} \ddot{x}_1 \\ \ddot{x}_2 \end{Bmatrix} + k \begin{bmatrix} 2 & -1 \\ -1 & 1 \end{bmatrix} \begin{Bmatrix} x_1 \\ x_2 \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \end{Bmatrix} \quad \text{Let } \lambda = \frac{m\omega^2}{k}$$

$$\begin{vmatrix} (2-3\lambda) & -1 \\ -1 & (1-\lambda) \end{vmatrix} = 0 \quad \lambda^2 - \frac{5}{3}\lambda + \frac{1}{3} = 0$$

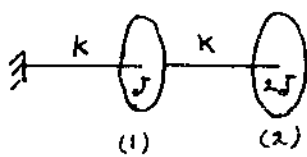
$$\text{Adj. matrix} = \begin{bmatrix} (1-\lambda) & 1 \\ 1 & (2-3\lambda) \end{bmatrix}$$

$$\lambda = 0.8333 \pm \sqrt{.6944 - .3333} \\ = 0.8333 \pm 0.6010 = \begin{cases} 0.2323 \\ 1.4343 \end{cases}$$

$$\text{Subst. } \lambda_1 \text{ into either column for 1st mode} = \begin{Bmatrix} .7677 \\ 1.000 \end{Bmatrix}$$

$$\text{Subst } \lambda_2 \text{ into " " " 2nd mode} = \begin{Bmatrix} -.4343 \\ 1.000 \end{Bmatrix}$$

6-16



$$\begin{bmatrix} J & 0 \\ 0 & 2J \end{bmatrix} \begin{Bmatrix} \ddot{\theta}_1 \\ \ddot{\theta}_2 \end{Bmatrix} + \begin{bmatrix} 2K & -K \\ -K & K \end{bmatrix} \begin{Bmatrix} \theta_1 \\ \theta_2 \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \end{Bmatrix}$$

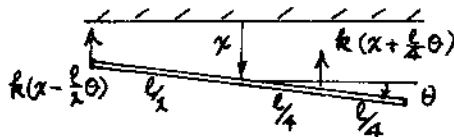
$$\text{Let } \lambda = \frac{\omega^2 J}{K}$$

$$\begin{vmatrix} (2-\lambda) & -1 \\ -1 & (1-2\lambda) \end{vmatrix} = 0 \quad 2\lambda^2 - 5\lambda + 1 = 0$$

$$\lambda = \frac{5}{4} \pm \sqrt{\frac{25}{16} - \frac{8}{16}} = \begin{cases} 0.2192 \\ 2.2808 \end{cases}$$

$$\text{Adj. matrix} = \begin{bmatrix} (1-2\lambda) & 1 \\ 1 & (2-\lambda) \end{bmatrix} \quad \begin{Bmatrix} \theta_1 \\ \theta_2 \end{Bmatrix}_1 = \begin{Bmatrix} 1.000 \\ 1.781 \end{Bmatrix} \quad \begin{Bmatrix} \theta_1 \\ \theta_2 \end{Bmatrix}_2 = \begin{Bmatrix} 1.00 \\ -2.2808 \end{Bmatrix}$$

6-17



$$m \frac{l^2}{12} \ddot{\theta} = \frac{l}{2} k(x - \frac{l}{2}\theta) - \frac{l}{4} k(x + \frac{l}{2}\theta) = \frac{kl}{4} x - \frac{5kl^2}{16} \theta$$

$$m \ddot{x} = -k(x - \frac{l}{2}\theta) - k(x + \frac{l}{2}\theta) = -2kx + \frac{kl}{2} \theta$$

$$m \begin{bmatrix} \frac{l^2}{12} & 0 \\ 0 & 1 \end{bmatrix} \begin{Bmatrix} \ddot{\theta} \\ \ddot{x} \end{Bmatrix} + k \begin{bmatrix} \frac{5l^2}{16} & -\frac{l}{4} \\ -\frac{l}{4} & 2 \end{bmatrix} \begin{Bmatrix} \theta \\ x \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \end{Bmatrix} \quad \text{Let } \lambda = \frac{\omega^2 m}{k}$$

$$\begin{vmatrix} (\frac{5l^2}{16} - \frac{l^2}{12}\lambda) - \frac{l}{4} \\ -\frac{l}{4} & (2-\lambda) \end{vmatrix} = 0$$

$$\lambda^2 - \frac{23}{4} \lambda + \frac{9 \times 12}{16} = 0$$

$$\lambda = \frac{23}{8} \pm \sqrt{\frac{97}{16}}$$

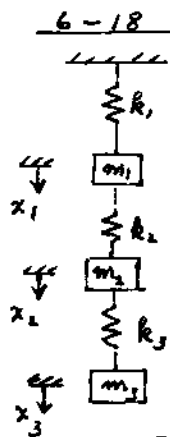
$$\lambda = \frac{1}{8} (23 \pm 9.849) = \begin{cases} 1.644 \\ 4.106 \end{cases} \quad \frac{l\theta}{x} = 4(2-\lambda) = \begin{cases} 1.424 \\ -8.424 \end{cases}$$

$$\begin{Bmatrix} \theta \\ x \end{Bmatrix}_1 = \begin{Bmatrix} 1.424/l \\ 1.000 \end{Bmatrix} \quad \begin{Bmatrix} \theta \\ x \end{Bmatrix}_2 = \begin{Bmatrix} -8.424/l \\ 1.000 \end{Bmatrix}$$

$$P = \begin{bmatrix} 1.424/l & -8.424/l \\ 1.000 & 1.000 \end{bmatrix} \quad \text{To show that } P' K P = \text{diagonal}$$

$$\begin{bmatrix} 1.424/l & 1.0 \\ -8.424/l & 1.0 \end{bmatrix} \begin{bmatrix} \frac{5l^2}{16} & -\frac{l}{4} \\ -\frac{l}{4} & 2 \end{bmatrix} \begin{bmatrix} 1.424/l & -8.424/l \\ 1.000 & 1.000 \end{bmatrix} = \begin{bmatrix} 1.424/l & 1.0 \\ -8.424/l & 1.0 \end{bmatrix} \begin{bmatrix} 1.952 & -2.8825 \\ 1.644 & 4.106 \end{bmatrix}$$

$$= \begin{bmatrix} 1.9217 & 0.0013 \\ 0.0013 & 28.388 \end{bmatrix} \approx \begin{bmatrix} 1.9217 & 0 \\ 0 & 28.388 \end{bmatrix}$$



Place unit load at 1.

$$a_{11} = \frac{1}{k_1} = a_{21} = a_{31} = a_{12} = a_{13}$$

Place unit load at 2

$$a_{22} = \left(\frac{1}{k_1} + \frac{1}{k_2}\right) = a_{32} = a_{23}$$

Place unit load at 3

$$a_{33} = \left(\frac{1}{k_1} + \frac{1}{k_2} + \frac{1}{k_3}\right)$$

Eq. of motion:

$$\begin{Bmatrix} x_1 \\ x_2 \\ x_3 \end{Bmatrix} = \omega^2 \begin{bmatrix} a_{11} & a_{12} & a_{13} \\ a_{21} & a_{22} & a_{23} \\ a_{31} & a_{32} & a_{33} \end{bmatrix} \begin{bmatrix} m_1 & 0 & 0 \\ 0 & m_2 & 0 \\ 0 & 0 & m_3 \end{bmatrix} \begin{Bmatrix} x_1 \\ x_2 \\ x_3 \end{Bmatrix}$$

6-19

$$\begin{bmatrix} m & 0 \\ 0 & 2m \end{bmatrix} \begin{Bmatrix} \ddot{x}_1 \\ \ddot{x}_2 \end{Bmatrix} + \begin{bmatrix} 2k & -k \\ -k & 2k \end{bmatrix} \begin{Bmatrix} x_1 \\ x_2 \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \end{Bmatrix} \quad \text{let } \lambda = \frac{m\omega^2}{k}$$

$$\begin{vmatrix} (2-\lambda) & -1 \\ -1 & (2-2\lambda) \end{vmatrix} = 0 \quad \lambda^2 - 3\lambda + \frac{3}{2} = 0$$

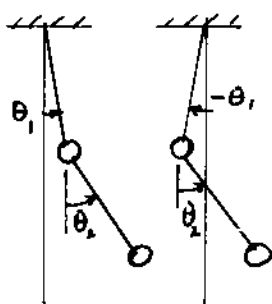
$$\lambda = \frac{3}{2} \pm \sqrt{\frac{9}{4} - \frac{3}{2}} = 1.5 \pm 0.866 = \begin{cases} 0.634 \\ 2.366 \end{cases}$$

$$\frac{x_1}{x_2} = \frac{2(1-\lambda)}{1} = \begin{cases} 0.732 \\ -2.732 \end{cases} \quad \therefore P = \begin{bmatrix} 0.732 & -2.732 \\ 1.000 & 1.000 \end{bmatrix}$$

$$P' M P \ddot{y} + P' K P y = 0$$

$$\begin{bmatrix} 2.535 & 0 \\ 0 & 9.48 \end{bmatrix} \begin{Bmatrix} \ddot{y}_1 \\ \ddot{y}_2 \end{Bmatrix} + \begin{bmatrix} 1.606 & 0 \\ 0 & 22.33 \end{bmatrix} \begin{Bmatrix} y_1 \\ y_2 \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \end{Bmatrix} \quad \text{uncoupled.}$$

6-20



$$\frac{l}{g} \begin{bmatrix} 2 & 1 \\ 1 & 1 \end{bmatrix} \begin{Bmatrix} \ddot{\theta}_1 \\ \ddot{\theta}_2 \end{Bmatrix} + \begin{bmatrix} 2 & 0 \\ 0 & 1 \end{bmatrix} \begin{Bmatrix} \theta_1 \\ \theta_2 \end{Bmatrix} = 0$$

\therefore dynamically coupled.

From Prob. 5-10

$$\begin{Bmatrix} \theta_1 \\ \theta_2 \end{Bmatrix}_1 = \begin{Bmatrix} 0.707 \\ 1.00 \end{Bmatrix} \quad \begin{Bmatrix} \theta_1 \\ \theta_2 \end{Bmatrix}_2 = \begin{Bmatrix} -0.707 \\ 1.00 \end{Bmatrix}$$

6-20 Cont.

Gen. mass

$$\text{mode 1} \quad (.707 \quad 1.0) \begin{bmatrix} 2 & 1 \\ 1 & 1 \end{bmatrix} \begin{Bmatrix} .707 \\ 1.0 \end{Bmatrix} = 3.414$$

$$\text{mode 2} \quad (-.707 \quad 1.0) \begin{bmatrix} 2 & 1 \\ 1 & 1 \end{bmatrix} \begin{Bmatrix} -.707 \\ 1.00 \end{Bmatrix} = 0.586$$

$$\tilde{P} = \begin{bmatrix} .707 & -.707 \\ 1.0 & 1.0 \end{bmatrix}$$

$$\tilde{P} = \begin{bmatrix} .207 & -1.207 \\ .293 & 1.707 \end{bmatrix}$$

$$\tilde{P}' M \tilde{P} = \begin{bmatrix} .207 & .293 \\ -1.207 & 1.707 \end{bmatrix} \begin{bmatrix} 2 & 1 \\ 1 & 1 \end{bmatrix} \begin{bmatrix} .207 & -1.207 \\ .293 & 1.707 \end{bmatrix} = \begin{bmatrix} 0.293 & 0 \\ 0 & 1.707 \end{bmatrix}$$

$$\tilde{P}' K \tilde{P} = \begin{bmatrix} & \\ & \end{bmatrix} \begin{bmatrix} 2 & 0 \\ 0 & 1 \end{bmatrix} \begin{bmatrix} & \\ & \end{bmatrix} = \begin{bmatrix} .1714 & 0 \\ 0 & 5.82 \end{bmatrix}$$

Normalized eq. of motion becomes

$$\begin{bmatrix} .293 & 0 \\ 0 & 1.707 \end{bmatrix} \begin{Bmatrix} \ddot{y}_1 \\ \ddot{y}_2 \end{Bmatrix} + \begin{bmatrix} .1714 & 0 \\ 0 & 5.82 \end{bmatrix} \begin{Bmatrix} y_1 \\ y_2 \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \end{Bmatrix} \quad \therefore \text{uncoupled}$$

6-21

see Prob. 6-11

$$\begin{bmatrix} (m_1 + m_2) & 0 & 0 \\ 0 & J_1 & 0 \\ 0 & 0 & J_2 \end{bmatrix} \begin{Bmatrix} \ddot{x} \\ \ddot{\theta}_1 \\ \ddot{\theta}_2 \end{Bmatrix} + \begin{bmatrix} \frac{24EI_1}{\ell_1^3} & -\frac{6EI_1}{\ell_1^2} & -\frac{6EI_1}{\ell_1^2} \\ -\frac{6EI_1}{\ell_1^2} & (\frac{4EI_1}{\ell_1} + \frac{4EI_2}{\ell_2}) & \frac{2EI_2}{\ell_2} \\ -\frac{6EI_1}{\ell_1^2} & \frac{2EI_2}{\ell_2} & (\frac{4EI_1}{\ell_1} + \frac{4EI_2}{\ell_2}) \end{bmatrix} \begin{Bmatrix} x \\ \theta_1 \\ \theta_2 \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \\ 0 \end{Bmatrix}$$

Assign numbers before solving.

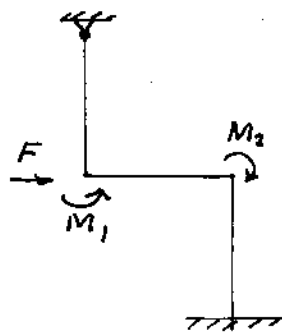
6-22

see Prob 6-12

$$\begin{bmatrix} (m_1 + m_2) & 0 & 0 \\ 0 & J_1 & 0 \\ 0 & 0 & J_2 \end{bmatrix} \begin{Bmatrix} \ddot{x} \\ \ddot{\theta}_1 \\ \ddot{\theta}_2 \end{Bmatrix} + \begin{bmatrix} \frac{6EI}{\ell^3} & -\frac{3EI}{\ell^2} & -\frac{3EI}{\ell^2} \\ -\frac{3EI}{\ell^2} & \frac{7EI}{\ell} & -\frac{2EI}{\ell} \\ -\frac{3EI}{\ell^2} & -\frac{2EI}{\ell} & \frac{7EI}{\ell} \end{bmatrix} \begin{Bmatrix} x \\ \theta_1 \\ \theta_2 \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \\ 0 \end{Bmatrix}$$

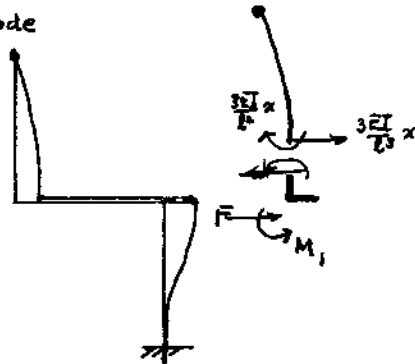
Assign numbers before solving

6-23



The problem is the superposition of the three modes below. Use Table 6.3-1 and examine the free-body-diagrams of members and corners.

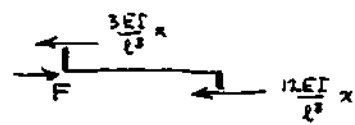
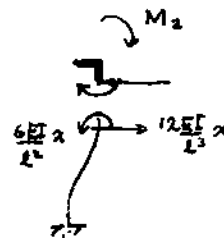
x -mode



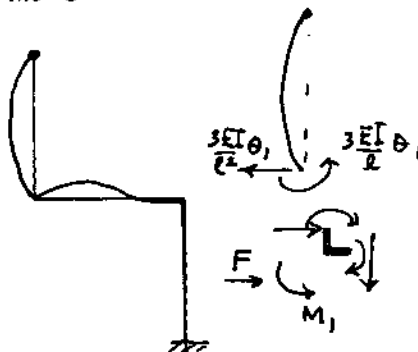
$$F = \frac{15EI}{l^3} x$$

$$M_1 = -\frac{3EI}{l^2} x$$

$$M_2 = \frac{6EI}{l^2} x$$



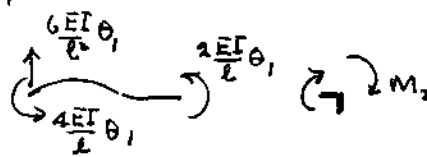
θ_1 -mode



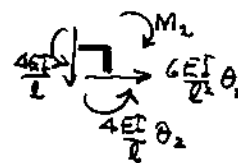
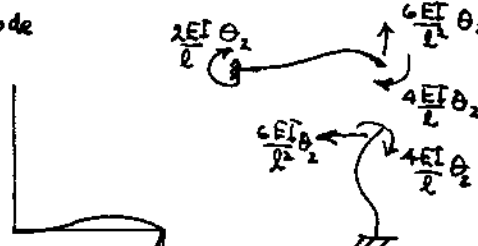
$$F = -\frac{3EI}{l^2} \theta_1$$

$$M_1 = \frac{7EI}{l} \theta_1$$

$$M_2 = -\frac{2EI}{l} \theta_1$$



θ_2 -mode



$$\begin{Bmatrix} F \\ M_1 \\ M_2 \end{Bmatrix} = \begin{bmatrix} \frac{15EI}{l^3} & -\frac{3EI}{l^2} & \frac{6EI}{l^2} \\ -\frac{3EI}{l^2} & \frac{7EI}{l} & -\frac{2EI}{l} \\ \frac{6EI}{l^2} & -\frac{2EI}{l} & \frac{8EI}{l} \end{bmatrix} \begin{Bmatrix} x \\ \theta_1 \\ \theta_2 \end{Bmatrix}$$

6-24

$$m \begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix} \begin{Bmatrix} \ddot{x}_1 \\ \ddot{x}_2 \end{Bmatrix} + c \begin{bmatrix} 2 & -1 \\ -1 & 1 \end{bmatrix} \begin{Bmatrix} \dot{x}_1 \\ \dot{x}_2 \end{Bmatrix} + k \begin{bmatrix} 2 & -1 \\ -1 & 2 \end{bmatrix} \begin{Bmatrix} x_1 \\ x_2 \end{Bmatrix} = \begin{Bmatrix} F_0 \\ 0 \end{Bmatrix} \sin \omega t$$

\therefore Damping is not proportional

6-25

char. eq. $\begin{vmatrix} (2-\lambda) & -1 \\ -1 & (2-\lambda) \end{vmatrix} = 0$ $\lambda = \begin{Bmatrix} 1 \\ 3 \end{Bmatrix}$ $\begin{Bmatrix} x_1 \\ x_2 \end{Bmatrix}_1 = \begin{Bmatrix} 1 \\ 1 \end{Bmatrix}$
 $\lambda = \frac{\omega^2 m}{k}$

$\therefore P = \begin{bmatrix} 1 & -1 \\ 1 & 1 \end{bmatrix}$ $\tilde{P} = \frac{1}{\sqrt{2m}} \begin{bmatrix} 1 & -1 \\ 1 & 1 \end{bmatrix}$ $\begin{Bmatrix} x_1 \\ x_2 \end{Bmatrix}_2 = \begin{Bmatrix} -1 \\ 1 \end{Bmatrix}$

Let $x = \tilde{P} y$ in eq. Prob. 6-24 and premult. by \tilde{P}'

$$\tilde{P}' M \tilde{P} \ddot{y} + \tilde{P}' C \tilde{P} \dot{y} + \tilde{P}' K \tilde{P} y = \tilde{P}' F$$

$$\begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix} \begin{Bmatrix} \ddot{y}_1 \\ \ddot{y}_2 \end{Bmatrix} + \frac{c}{2m} \begin{bmatrix} 1 & -1 \\ -1 & 5 \end{bmatrix} \begin{Bmatrix} \dot{y}_1 \\ \dot{y}_2 \end{Bmatrix} + \frac{k}{m} \begin{bmatrix} 1 & 0 \\ 0 & 3 \end{bmatrix} \begin{Bmatrix} y_1 \\ y_2 \end{Bmatrix} = \begin{Bmatrix} F_0 \\ -F_0 \end{Bmatrix} \frac{\sin \omega t}{\sqrt{2m}}$$

\therefore coupled only by damping

$$\ddot{y}_1 + \frac{c}{2m} (\dot{y}_1 - \dot{y}_2) + \frac{k}{m} y_1 = \frac{F_0}{\sqrt{2m}} \sin \omega t$$

$$\ddot{y}_2 + \frac{c}{2m} (-\dot{y}_1 + 5\dot{y}_2) + 3\frac{k}{m} y_2 = \frac{-F_0}{\sqrt{2m}} \sin \omega t$$

Steady State solution

$$\left(\frac{k}{m} - \omega^2 + i \frac{c}{2m} \omega \right) Y_1 - i \left(\frac{c}{2m} \omega \right) Y_2 = \frac{F_0}{\sqrt{2m}}$$

$$-i \left(\frac{c}{2m} \omega \right) Y_1 + \left(\frac{3k}{m} - \omega^2 + i \frac{5c}{2m} \omega \right) Y_2 = \frac{-F_0}{\sqrt{2m}}$$

$$Y_1 = \frac{\begin{vmatrix} 1 & i\omega \frac{c}{2m} \\ -1 & \left(\frac{3k}{m} - \omega^2 + i\omega \frac{5c}{2m} \right) \end{vmatrix} \frac{F_0}{\sqrt{2m}}}{\left(\frac{k}{m} - \omega^2 + i \frac{c}{2m} \omega \right) \left(\frac{3k}{m} - \omega^2 + i \omega \frac{5c}{2m} \right) + \left(\frac{\omega c}{2m} \right)^2}$$

6-26

Let. $\omega_0^2 = \frac{k}{m}$, $\alpha = \frac{k_1}{C}$, $\beta = \frac{k_1}{m}$

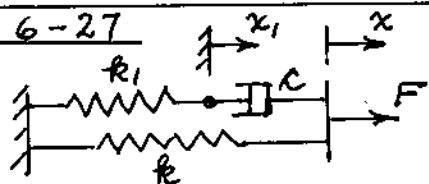
The equations are rewritten as

$$\ddot{x} = -\omega_0^2 x - \beta x_1 + \frac{F}{m}$$

$$\dot{x}_1 = \dot{x} - \alpha x_1$$

Let $\begin{cases} x_1 = z_1 \\ \dot{x}_1 = \dot{z}_1 \\ x = z_2 \\ \dot{x} = z_3 = \dot{z}_2 \end{cases}$ then $\begin{cases} \dot{z}_1 \\ \dot{z}_2 \\ \dot{z}_3 \end{cases} = \begin{bmatrix} -\alpha & 0 & 1 \\ 0 & 0 & 1 \\ -\beta & -\omega_0^2 & 0 \end{bmatrix} \begin{cases} z_1 \\ z_2 \\ z_3 \end{cases} + \begin{cases} 0 \\ 0 \\ \frac{F}{m} \end{cases}$

6-27



$$F = kx + C(\dot{x} - \dot{x}_1) \quad (a)$$

$$k_1 x_1 = C(\dot{x} - \dot{x}_1) \quad (b)$$

Assume F to be harmonic; then from (b)

$$x_1 = \frac{i\omega C}{k_1 + i\omega C} x = \frac{i(\omega C/k_1)}{1 + i(\omega C/k_1)} x$$

Subst. into (a)

$$\begin{aligned} F &= kx + i\omega C \left[1 - \frac{i(\omega C/k_1)}{1 + i(\omega C/k_1)} \right] x \\ &= \frac{[k(1 + \frac{i\omega C}{k_1}) + i\omega C]}{(1 + \frac{i\omega C}{k_1})} \cdot \frac{(1 - \frac{i\omega C}{k_1})}{(1 - \frac{i\omega C}{k_1})} x \\ &= \left\{ \frac{k + (k+k_1)(\frac{\omega C}{k_1})^2}{1 + (\frac{\omega C}{k_1})^2} + \frac{i\omega C}{1 + (\frac{\omega C}{k_1})^2} \right\} x \\ &= \left\{ k_{eq} + i\omega C_{eq} \right\} x \end{aligned}$$

6-28

Refer to Prob. 6-16

$$X_1 = \begin{Bmatrix} \theta_1 \\ \theta_2 \end{Bmatrix}_1 = \begin{Bmatrix} 1.00 \\ 1.781 \end{Bmatrix}$$

$$X_2 = \begin{Bmatrix} \theta_1 \\ \theta_2 \end{Bmatrix}_2 = \begin{Bmatrix} 1.00 \\ -0.2808 \end{Bmatrix}$$

$$\begin{aligned} X_1' K X_2 &= (1.00 \quad 1.781) K \begin{bmatrix} 2 & -1 \\ -1 & 1 \end{bmatrix} \begin{Bmatrix} 1.00 \\ -0.2808 \end{Bmatrix} \\ &= (1.00 \quad 1.781) \begin{Bmatrix} 2.2808 \\ -1.2808 \end{Bmatrix} = 2.2808 - 2.2811 \\ &= -0.0003 \approx 0 \end{aligned}$$

6-29
 $\phi = \text{normal mode}$
 $\times \text{ by } KM^{-1}$

$$K\phi_s = \omega_s^2 M\phi_s$$

$$KM^{-1}K\phi_s = \omega_s^2 KM^{-1}M\phi_s = \omega_s^2 K\phi_s$$

$$\underline{\underline{\phi_n' [KM^{-1}K]\phi_s = \omega_s^2 [\phi_n' K\phi_s] = 0 \quad \text{for } n \neq s}}$$

$$\text{Repeat } KM^{-1}K\phi_s = \omega_s^2 K\phi_s \quad \times \text{ by } KM^{-1}$$

$$KM^{-1}KM^{-1}K\phi_s = \omega_s^2 [KM^{-1}K]\phi_s \quad \times \text{ by } \phi_n'$$

$$\underline{\underline{\phi_n' [KM^{-1}]^2 K\phi_s = \omega_s^2 \phi_n' [KM^{-1}K]\phi_s = 0 \quad n \neq s}}$$

$$\text{Repeat to obtain } \phi_n' [KM^{-1}]^h K\phi_s = 0$$

6-30

$$K\phi_s = \omega_s^2 M\phi_s, \quad MK^{-1}K\phi_s = \omega_s^2 MK^{-1}M\phi_s$$

$$\therefore M\phi_s = \omega_s^2 MK^{-1}M\phi_s \quad \text{ie } K^{-1}K = I$$

$$\phi_n' M\phi_s = \omega_s^2 \phi_n' [MK^{-1}M]\phi_s = 0 \quad \text{ie } \phi_n' M\phi_s = 0 \quad n \neq s$$

Repeat

$$MK^{-1}M\phi_s = \omega_s^2 [MK^{-1}]^2 M\phi_s$$

$$\underline{\underline{\phi_n' MK^{-1}M\phi_s = \omega_s^2 \phi_n' [MK^{-1}]^2 M\phi_s = 0 \quad n \neq s}}$$

Repeat h times.

6-31 Refer to Ex. 6.9-1 for ϕ_i & ω_i

For second mode

$$m_{22} \ddot{q}_2 + c_{22} \dot{q}_2 + k_{22} q_2 = -\ddot{u}_0(t) \sum_{i=1}^{10} m_i \phi_2(x_i)$$

$$m_{22} = \sum_{i=1}^{10} m_i \phi_2^2(x_i) = 5.5235 m$$

$$c_{22} = 2 \zeta_2 \omega_2 m_{22} = 2 \zeta_2 (0.4451 \sqrt{\frac{k}{m}}) m_{22} = 0.8902 \zeta_2 \sqrt{\frac{k}{m}} m_{22}$$

$$k_{22} = \omega_2^2 m_{22} = (0.1981 \frac{k}{m}) m_{22}$$

$$\sum_{i=1}^{10} m_i \phi_2(x_i) = -2.2470 m$$

$$\therefore \ddot{q}_2 + 0.8902 \zeta_2 \sqrt{\frac{k}{m}} \dot{q}_2 + 0.1981 \frac{k}{m} q_2 = \frac{2.247}{5.5235} \ddot{u}_0(t) \\ = 0.4068 \ddot{u}_0(t)$$

For 3rd mode

$$m_{33} = 8.5957 m, \quad c_{33}/m_{33} = 2 \zeta_3 (1.7307 \sqrt{\frac{k}{m}}) = 1.4614 \zeta_3 \sqrt{\frac{k}{m}}$$

$$k_{33} = 0.5339 \frac{k}{m}, \quad \sum_{i=1}^{10} m_i \phi_3(x_i) = 2.8095 m$$

$$\ddot{q}_3 + 1.4614 \zeta_3 \sqrt{\frac{k}{m}} \dot{q}_3 + 0.5339 \frac{k}{m} q_3 = -0.3268 \ddot{u}_0(t)$$

6-32 From Ex. 6.9-1

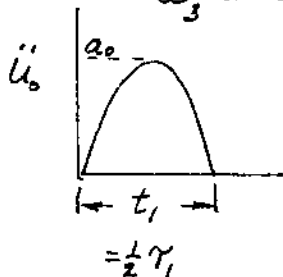
$$\ddot{q}_1 + 0.299 \zeta_1 \sqrt{\frac{k}{m}} \dot{q}_1 + 0.02235 \frac{k}{m} q_1 = -1.2672 \ddot{u}_0(t)$$

$$\omega_1^2 = 0.02235 \frac{k}{m}, \quad \omega_1 = 0.1495 \sqrt{\frac{k}{m}} = \frac{2\pi}{\tau_1}$$

$$\therefore \tau_1 = \frac{2\pi}{0.1495} \sqrt{\frac{m}{k}} = 42.028 \sqrt{\frac{m}{k}}$$

$$\omega_2 = \frac{2\pi}{\gamma_2} = 0.4451 \sqrt{\frac{k}{m}}, \quad \gamma_2 = 14.1168 \sqrt{\frac{m}{k}}$$

$$\omega_3 = 0.7307 \sqrt{\frac{k}{m}}, \quad \gamma_3 = 8.5989 \sqrt{\frac{m}{k}}$$



See Fig 4.4-3 for shock spectrum

$$\frac{t_1}{\gamma_1} = 0.50 \quad \left(\frac{xk}{F_0} \right)_{\max} = 1.5$$

$$\frac{t_1}{\gamma_1} \frac{\gamma_1}{\gamma_2} = \frac{t_1}{\gamma_2} = .5 \frac{42.028}{14.116} = 1.4886 \therefore \left(\frac{xk}{F_0} \right)_{\max} = 1.5$$

$$\frac{t_1}{\gamma_3} = .50 \frac{42.028}{8.5989} = 2.4438 \therefore \left(\frac{xk}{F_0} \right)_{\max} = 1.13$$

The right side of the DEs are:

$$-\ddot{u}_0(t) \frac{\sum m \phi_i^2}{\sum m \phi_i^2} = -1.2672 \ddot{u}_0 = \frac{F_0}{m} \quad \text{by comparison}$$

with $\ddot{q} + 25\omega_n \dot{q} + \omega_n^2 q = \frac{F_0}{m}$

$$\therefore F_0 = -1.2672 m a_0 \quad \therefore \text{in place of } \left(\frac{xk}{F_0} \right)_{\max}$$

$$\text{we use } \left(\frac{qk}{F_0} \right)_{\max} = \frac{qk}{-1.2672 m a_0} \quad \text{for mode 1.} = 1.5$$

$$\therefore (q_1)_{\max} = -1.5 \times 1.2672 \frac{m}{k} a_0 = \underline{\underline{-1.9008 \frac{m a_0}{k}}}$$

Similarly for 2nd mode & 3rd mode

$$(q_2)_{\max} = 1.5 \times .4068 \frac{m a_0}{k} = \underline{\underline{0.6102 \frac{m a_0}{k}}}$$

$$(q_3)_{\max} = 1.13 \times (-.3263) \frac{m a_0}{k} = \underline{\underline{-0.3693 \frac{m a_0}{k}}}$$

6-32 Cont:

$$x(t) = \phi_1 q_1 + \phi_2 q_2 + \phi_3 q_3 + \dots$$

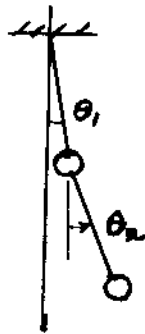
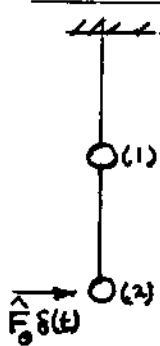
but at 10th floor $\phi_i = 1.0$

$$\therefore x(t) = q_1 + q_2 + q_3$$

From Eq. (6.9-6)

$$\begin{aligned} |x(10)|_{\max} &= (q_1)_{\max} + \sqrt{(q_2)_{\max}^2 + (q_3)_{\max}^2} \\ &= 1.90 + \sqrt{.610^2 + .369^2} \\ &= 1.90 + 0.711 = \underline{\underline{2.61 \frac{m a_0}{k}}} \end{aligned}$$

6-33



Impulse = change in momentum

At $t=0$ mass at (2) will acquire a velocity of $\frac{\hat{F}_0}{m} = v(0) = l \dot{\theta}_2(0)$

$$\therefore \dot{\theta}_2(0) = \frac{\hat{F}}{ml} \quad \dot{\theta}_1(0) = 0$$

$$\theta = \phi_1 q_1 + \phi_2 q_2$$

$$\therefore \begin{Bmatrix} \theta_1 \\ \theta_2 \end{Bmatrix} = \begin{Bmatrix} .707 \\ 1.00 \end{Bmatrix} q_1 + \begin{Bmatrix} -.707 \\ 1.00 \end{Bmatrix} q_2$$

$$= \begin{Bmatrix} .707 \\ 1.00 \end{Bmatrix} A \sin .764 \sqrt{\frac{g}{l}} t + \begin{Bmatrix} -.707 \\ 1.00 \end{Bmatrix} B \sin 1.85 \sqrt{\frac{g}{l}} t$$

$$\sqrt{\frac{l}{g}} \begin{Bmatrix} \dot{\theta}_1 \\ \dot{\theta}_2 \end{Bmatrix} = .764 \begin{Bmatrix} .707 \\ 1.00 \end{Bmatrix} A \cos .764 \sqrt{\frac{g}{l}} t + 1.85 \begin{Bmatrix} -.707 \\ 1.00 \end{Bmatrix} B \cos 1.85 \sqrt{\frac{g}{l}} t$$

At $t = 0$

$$\sqrt{\frac{l}{g}} \begin{Bmatrix} 0 \\ \frac{\hat{F}}{ml} \end{Bmatrix} = .764 \begin{Bmatrix} .707 \\ 1.00 \end{Bmatrix} A + 1.85 \begin{Bmatrix} -.707 \\ 1.00 \end{Bmatrix} B$$

6-33 Cont

$$\therefore 0 = (.764)(.707)A - (1.85)(.707)B$$

$$B = 0.413 A$$

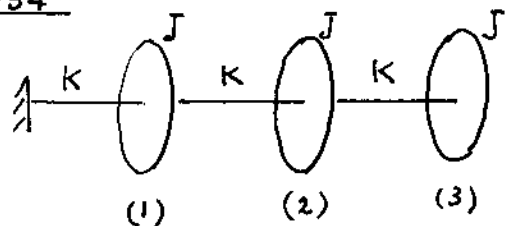
$$\sqrt{\frac{l}{g}} \frac{\hat{F}_0}{m l} = .764 A + 1.85 (.413 A) = 1.528 A$$

$$\therefore A = 0.6544 \sqrt{\frac{l}{g}} \frac{\hat{F}}{m l}$$

$$B = 0.2703 \sqrt{\frac{l}{g}} \frac{\hat{F}}{m l}$$

$$\begin{Bmatrix} \theta_1 \\ \theta_2 \end{Bmatrix} = \sqrt{\frac{l}{g}} \frac{\hat{F}}{m l} \left[(0.6544) \begin{Bmatrix} .707 \\ 1.00 \end{Bmatrix} \sin .764 \sqrt{\frac{g}{l}} t + (.2703) \begin{Bmatrix} -.707 \\ 1.00 \end{Bmatrix} \sin 1.85 \sqrt{\frac{g}{l}} t \right]$$

6-34



$$\lambda = \frac{J \omega^2}{K}$$

$$\phi = \begin{Bmatrix} \theta_1 \\ \theta_2 \\ \theta_3 \end{Bmatrix}$$

$$J \begin{bmatrix} 1 & 0 & 0 \\ 0 & 1 & 0 \\ 0 & 0 & 1 \end{bmatrix} \begin{Bmatrix} \ddot{\theta}_1 \\ \ddot{\theta}_2 \\ \ddot{\theta}_3 \end{Bmatrix} + K \begin{bmatrix} 2 & -1 & 0 \\ -1 & 2 & -1 \\ 0 & -1 & 1 \end{bmatrix} \begin{Bmatrix} \theta_1 \\ \theta_2 \\ \theta_3 \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \\ M_0 u(t) \end{Bmatrix}$$

Let $\begin{Bmatrix} \theta_1 \\ \theta_2 \\ \theta_3 \end{Bmatrix} = P \begin{Bmatrix} q_1 \\ q_2 \\ q_3 \end{Bmatrix}$ & premultiply by P'

$$P' J P \ddot{q} + P' K P q = P' \{M\}$$

$$m_{ii} \ddot{q}_i + k_{ii} q_i = M_0 u(t) \theta_{3i}$$

$$\ddot{q}_i + \omega_i^2 q_i = \frac{M_0 (\theta_3)_i}{m_{ii}} u(t)$$

$$m_{11} = \phi'_1 J \phi_1 = J (.328^2 + .591^2 + .737^2) = 1.00 J$$

$$\omega_1^2 = .198 \frac{K}{J}$$

$$m_{22} = J (.737^2 + .328^2 + .591^2) = 1.00 J$$

$$\omega_2^2 = 1.555 \frac{K}{J}$$

$$m_{33} = 1.00 J$$

$$\omega_3^2 = 3.247 \frac{K}{J}$$

Right side of eq $\frac{M_0 (\theta_3)_i}{m_{ii}} u(t) =$

$$\frac{.737}{J} M_0 u(t)$$

$$- \frac{.591}{J} M_0 u(t)$$

and

$$\frac{.328}{J} M_0 u(t)$$

6-34 Cont. \therefore Modal eqs. are

$$\left. \begin{aligned} \ddot{q}_1 + (.198 \frac{K}{J}) q_1 &= \frac{.737}{J} M_0 u(t) \\ \ddot{q}_2 + (1.555 \frac{K}{J}) q_2 &= -\frac{.591}{J} M_0 u(t) \\ \ddot{q}_3 + (3.247 \frac{K}{J}) q_3 &= \frac{.328}{J} M_0 u(t) \end{aligned} \right\} \begin{aligned} &\text{For } u(t) \text{ the max} \\ &\text{response} = 2 \times \frac{F_0}{k} \text{ if} \\ &DE = \ddot{x} + \omega_n^2 x = \frac{F_0}{m} \\ &\text{Sol: } x = \frac{F_0}{k} (1 - \cos \omega_n t) \end{aligned}$$

$$\therefore q_1(t) = \frac{.737 M_0}{\sqrt{(.198 \frac{K}{J})}} (1 - \cos \sqrt{.198 \frac{K}{J}} t) = 3.72 \frac{M_0}{K} (1 - \cos \omega_1 t)$$

$$q_2(t) = \frac{-.591}{1.555} \frac{M_0}{K} (1 - \cos \omega_2 t) = -0.380 \frac{M_0}{K} (1 - \cos \omega_2 t)$$

$$q_3(t) = \frac{.328}{3.247} \frac{M_0}{K} (1 - \cos \omega_3 t) = 0.101 \frac{M_0}{K} (1 - \cos \omega_3 t)$$

$$\theta_3(t) = \phi_1(\theta_3) q_1(t) + \phi_2(\theta_3) q_2(t) + \phi_3(\theta_3) q_3(t)$$

$$= 0.737 q_1(t) - .591 q_2(t) + .328 q_3(t) = \text{superposition of above } q_i$$

By Shock Spectrum technique

$$\begin{aligned} |\theta_3(t)|_{\max} &= .737 q_{1,\max} + \sqrt{(.591 q_{2,\max})^2 + (.328 q_{3,\max})^2} \\ &= .737 (2 \times 3.72 \frac{M_0}{K}) + \sqrt{(.591 \times 2 \times .380 \frac{M_0}{K})^2 + (.328 \times 2 \times .101 \frac{M_0}{K})^2} \\ &= (5.483 + .454) \frac{M_0}{K} = \underline{\underline{5.937 \frac{M_0}{K}}} \end{aligned}$$

6-35

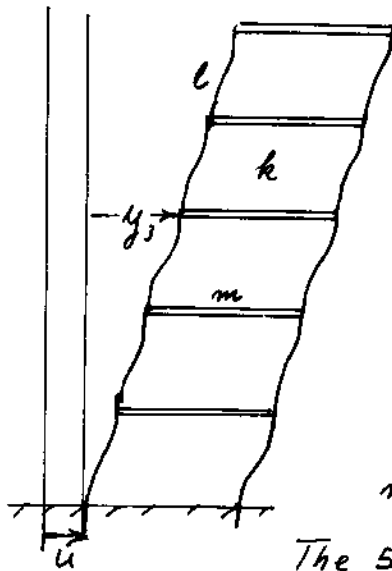
$$y(t) = \phi_1 q_1 + \phi_2 q_2 + \phi_3 q_3 + \phi_4 q_4 + \phi_5 q_5$$

1st Two Normal Modes obtained from Computer

$$\begin{aligned} \frac{m \omega_1^2}{k} &= 0.08101 & \begin{Bmatrix} .1699 \\ .3260 \\ .4557 \\ .5485 \\ .5969 \end{Bmatrix} &= \phi_1, & \frac{m \omega_2^2}{k} &= 0.6903 & \begin{Bmatrix} .4557 \\ .5969 \\ .3260 \\ -.1699 \\ -.5485 \end{Bmatrix} &= \phi_2 \\ \gamma_1 &= 22.08 \sqrt{\frac{m}{k}} & & & \gamma_2 &= 7.563 \sqrt{\frac{m}{k}} & & \end{aligned}$$

$$\frac{\sum m \phi_1}{\sum m \phi_1^2} = 2.097$$

$$\frac{\sum m \phi_2}{\sum m \phi_2^2} = 0.6602$$



For $K_R = \infty$, $\theta = 0$ and we have only translation of ground plus elastic translation of each floor. The more general case of $\theta \neq 0$ should be deferred until Ch 8, with Lagrange's Eq.

Write force eq. for one mass (say 3rd floor)

$$m(\ddot{u} + \ddot{y}_3) = -k(y_3 - y_2) + k(y_4 - y_3)$$

The 5 equations like the above can be written in matrix form

$$m \begin{bmatrix} 1 & 0 & 0 & 0 & 0 \\ 0 & 1 & 0 & 0 & 0 \\ 0 & 0 & 1 & 0 & 0 \\ 0 & 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 0 & 1 \end{bmatrix} \begin{Bmatrix} \ddot{y}_1 \\ \ddot{y}_2 \\ \ddot{y}_3 \\ \ddot{y}_4 \\ \ddot{y}_5 \end{Bmatrix} + k \begin{bmatrix} 2 & -1 & 0 & 0 & 0 \\ -1 & 2 & -1 & 0 & 0 \\ 0 & -1 & 2 & -1 & 0 \\ 0 & 0 & -1 & 2 & -1 \\ 0 & 0 & 0 & -1 & 1 \end{bmatrix} \begin{Bmatrix} y_1 \\ y_2 \\ y_3 \\ y_4 \\ y_5 \end{Bmatrix} = -m \begin{bmatrix} 1 & 0 & 0 & 0 & 0 \\ 0 & 1 & 0 & 0 & 0 \\ 0 & 0 & 1 & 0 & 0 \\ 0 & 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 0 & 1 \end{bmatrix} \ddot{u}$$

or

$$M \ddot{y} + K y = -M \ddot{u}$$

Let $y = \phi_1 q_1 + \phi_2 q_2$ & decouple by $P' K P$

$$\text{or } \begin{Bmatrix} y_1 \\ y_2 \end{Bmatrix} = \begin{bmatrix} \phi_1 & \phi_2 \end{bmatrix} \begin{Bmatrix} q_1 \\ q_2 \end{Bmatrix} = P q$$

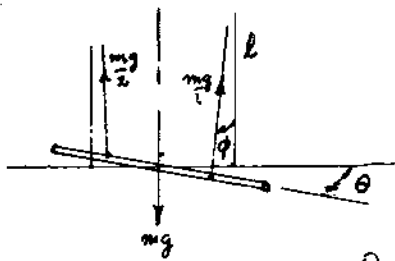
$$P' M P \ddot{q} + P' K P q = -P' M \ddot{u}$$

The modal eqs. become

$$\ddot{q}_1 + \omega_1^2 q_1 = -\frac{\sum m \phi_1}{\sum m \phi_1^2} \ddot{u}$$

$$\ddot{q}_2 + \omega_2^2 q_2 = -\frac{\sum m \phi_2}{\sum m \phi_2^2} \ddot{u}$$

6-36



$$\phi \approx \frac{a\theta}{2l}$$

Torsional Oscil.

$$J\ddot{\theta} = -2 \frac{mg}{2} \frac{a\theta}{2l} \frac{a}{2} = -mg \frac{a^2}{4l} \theta$$

$$\therefore \omega_T^2 = \frac{mg a^2}{4l J} = \frac{mg a^2}{4l \frac{m L^2}{12}} = 3 \frac{g}{L} \left(\frac{a}{L}\right)^2$$

Out-of plane oscillation is that of simple pendulum with $\omega^2 = \frac{g}{L}$. For $\omega^2 = \omega_T^2$

$$\frac{g}{L} = 3 \frac{g}{L} \left(\frac{a}{L}\right)^2 \quad \therefore \frac{a}{L} = \frac{1}{\sqrt{3}}$$

With small eccentricity to excite torsional oscil, there will be beating (see Prob. 1-4)

6-37

Modal damping given as $\zeta_1 = .05$, $\zeta_2 = .13$

From Eq. (6.8-9)

$$2\zeta_1 \omega_1 = \alpha + \beta \omega_1^2$$

$$2\zeta_2 \omega_2 = \alpha + \beta \omega_2^2$$

$$2(\zeta_1 \omega_1 - \zeta_2 \omega_2) = \beta(\omega_1^2 - \omega_2^2)$$

$$\therefore \beta = \frac{2(\zeta_2 \omega_2 - \zeta_1 \omega_1)}{\omega_2^2 - \omega_1^2}, \quad \alpha = \frac{2\omega_1 \omega_2 (\zeta_1 \omega_2 - \zeta_2 \omega_1)}{\omega_2^2 - \omega_1^2}$$

$$\therefore \beta = \frac{2(.05 \times .445 - .13 \times 1.247) \sqrt{\frac{k}{m}}}{(.198 - 1.555) \frac{k}{m}} = \underline{\underline{0.2061 \sqrt{\frac{m}{k}}}}$$

$$\alpha = \frac{2 \times .445 \times 1.247 (.05 \times 1.247 - .13 \times .445) \sqrt{\frac{k}{m}}}{1.357} = \underline{\underline{0.00370 \sqrt{\frac{k}{m}}}}$$

For 3rd mode

$$2\zeta_3 \omega_3 = \alpha + \beta \omega_3^2$$

$$\zeta_3 = \frac{\alpha + \beta \omega_3^2}{2\omega_3}$$

$$\zeta_3 = \frac{.00370 + .2061 \times 3.247}{2 \times 1.802} = \underline{\underline{0.1867}}$$

6-39

See Ex. 6.5-2

$$C_i = \frac{X_i' M u(0)}{X_i' M X_i}$$

From data given $X_i' M X_i = 1.0$ m.

$$\therefore C_1 = \frac{(.737 \ .591 \ .328)}{1.0} \begin{Bmatrix} .520 \\ -.100 \\ .205 \end{Bmatrix} = .3914$$

$$C_2 = (-.591 \ .328 \ .737) \begin{Bmatrix} .520 \\ -.100 \\ .205 \end{Bmatrix} = -.1890 = -.4829 C_1$$

$$C_3 = (.328 \ -.737 \ .591) \begin{Bmatrix} .520 \\ -.100 \\ .205 \end{Bmatrix} = .3654 = .9336 C_1$$

6-38

$$X_i' M X_j = (- \ - \ -)_i m [I] \begin{Bmatrix} - \\ - \\ - \end{Bmatrix}_j = m (- \ - \ -)_i \begin{Bmatrix} - \\ - \\ - \end{Bmatrix}_j$$

$$\text{With the modes given } X_i' M X_j = \begin{cases} 0 & \text{for } j \neq i \\ 1 & \text{for } j = i \end{cases}$$

6-40

$$X(t) = \sum_i X_i (A_i \sin \omega_i t + B_i \cos \omega_i t)$$

$$\dot{X}(t) = \sum_i \omega_i X_i (A_i \cos \omega_i t - B_i \sin \omega_i t)$$

$$\dot{X}(0) = \sum_i \omega_i X_i A_i$$

$$X_j' M \dot{X}(0) = \sum_i \omega_i X_j' M X_i A_i = \omega_j X_j' M X_j A_j$$

$$\therefore A_j = \frac{X_j' M \dot{X}(0)}{\omega_j X_j' M X_j}$$

$$B_j = \frac{X_j' M X(0)}{X_j' M X_j}$$

6-41 The bending of the shaft is measured from the slope of the bearing, which is β . Thus y and θ in Example 6.1-4 must first be replaced by η and $(\theta - \beta)$ respectively.

$$\begin{Bmatrix} \eta \\ \theta - \beta \end{Bmatrix} = \begin{bmatrix} a_{11} & a_{12} \\ a_{21} & a_{22} \end{bmatrix} \begin{Bmatrix} P \\ M \end{Bmatrix}$$

From geometry Fig P6-41

$$y = \frac{Pl}{K} + \beta l + \eta$$

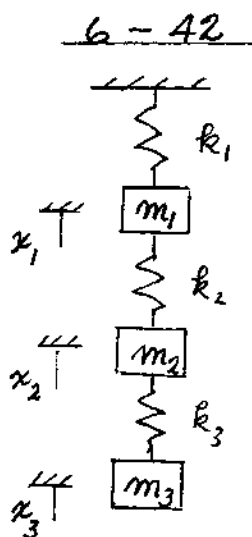
$$M_B = K\beta = Pl + M \quad \therefore \beta = \frac{Pl}{K} + \frac{M}{K}$$

$$\text{and } \eta = y - \frac{Pl}{K} - \beta l = y - \frac{Pl}{K} - \frac{Pl^2}{K} - \frac{Ml}{K}$$

$$\therefore y - \frac{Pl}{K} - \frac{Pl^2}{K} - \frac{Ml}{K} = a_{11}P + a_{12}M$$

$$\theta - \frac{Pl}{K} - \frac{M}{K} = a_{21}P + a_{22}M$$

$$\begin{Bmatrix} y \\ \theta \end{Bmatrix} = \begin{bmatrix} (a_{11} + \frac{1}{K} + \frac{l^2}{K}) & (a_{12} + \frac{l}{K}) \\ (a_{21} + \frac{l}{K}) & (a_{22} + \frac{1}{K}) \end{bmatrix} \begin{Bmatrix} P \\ M \end{Bmatrix} = \begin{bmatrix} \bar{a}_{11} & \bar{a}_{12} \\ \bar{a}_{21} & \bar{a}_{22} \end{bmatrix} \begin{Bmatrix} P \\ M \end{Bmatrix}$$



$$\begin{bmatrix} m_1 & 0 & 0 \\ 0 & m_2 & 0 \\ 0 & 0 & m_3 \end{bmatrix} \begin{Bmatrix} \ddot{x}_1 \\ \ddot{x}_2 \\ \ddot{x}_3 \end{Bmatrix} + \begin{bmatrix} (k_1 + k_2) & -k_2 & 0 \\ -k_2 & (k_2 + k_3) & -k_3 \\ 0 & -k_3 & k_3 \end{bmatrix} \begin{Bmatrix} x_1 \\ x_2 \\ x_3 \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \\ 0 \end{Bmatrix}$$

$$M\ddot{X} + KX = 0$$

$$[-\omega^2 I + M^{-1}K]X = 0$$

$$[A - \lambda I]X = 0$$

$$\text{where } A = M^{-1}K$$

6-42 Cont.

$$M^{-1} = \begin{bmatrix} \frac{1}{m_1} & 0 & 0 \\ 0 & \frac{1}{m_2} & 0 \\ 0 & 0 & \frac{1}{m_3} \end{bmatrix}$$

$$A = M^{-1} K = \begin{bmatrix} \frac{1}{m_1} & 0 & 0 \\ 0 & \frac{1}{m_2} & 0 \\ 0 & 0 & \frac{1}{m_3} \end{bmatrix} \begin{bmatrix} (k_1+k_2) & -k_2 & 0 \\ -k_2 & (k_2+k_3) & -k_3 \\ 0 & -k_3 & k_3 \end{bmatrix}$$

$$= \begin{bmatrix} \frac{1}{m_1}(k_1+k_2) & -\frac{1}{m_1}k_2 & 0 \\ -\frac{1}{m_2}k_2 & \frac{1}{m_2}(k_2+k_3) & -\frac{1}{m_2}k_3 \\ 0 & -\frac{1}{m_3}k_3 & \frac{1}{m_3}k_3 \end{bmatrix} \therefore \text{not symmetric}$$

Next Follow Footnote p165

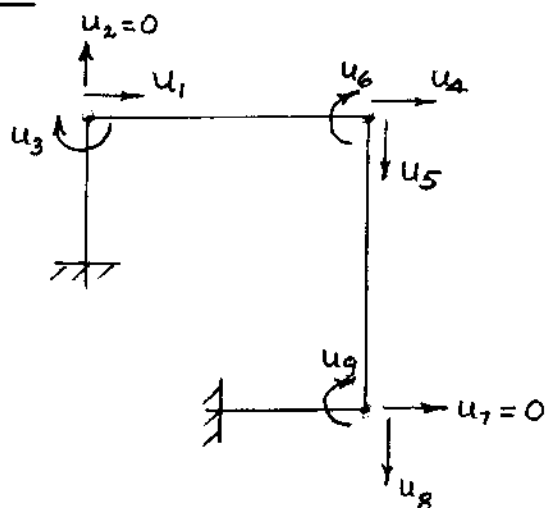
$$Q = M^{\frac{1}{2}} = \begin{bmatrix} \sqrt{m_1} & 0 & 0 \\ 0 & \sqrt{m_2} & 0 \\ 0 & 0 & \sqrt{m_3} \end{bmatrix} \quad Q^{-1} = \begin{bmatrix} \frac{1}{\sqrt{m_1}} & 0 & 0 \\ 0 & \frac{1}{\sqrt{m_2}} & 0 \\ 0 & 0 & \frac{1}{\sqrt{m_3}} \end{bmatrix} = Q^{-T}$$

$$\text{New } A = Q^{-T} K Q^{-1} = \begin{bmatrix} \frac{1}{\sqrt{m_1}} & 0 & 0 \\ 0 & \frac{1}{\sqrt{m_2}} & 0 \\ 0 & 0 & \frac{1}{\sqrt{m_3}} \end{bmatrix} \begin{bmatrix} (k_1+k_2) & -k_2 & 0 \\ -k_2 & (k_2+k_3) & -k_3 \\ 0 & -k_3 & k_3 \end{bmatrix} \begin{bmatrix} \frac{1}{\sqrt{m_1}} & 0 & 0 \\ 0 & \frac{1}{\sqrt{m_2}} & 0 \\ 0 & 0 & \frac{1}{\sqrt{m_3}} \end{bmatrix}$$

$$= \begin{bmatrix} \frac{1}{m_1}(k_1+k_2) & -\frac{1}{\sqrt{m_1 m_2}}k_2 & 0 \\ -\frac{1}{\sqrt{m_1 m_2}}k_2 & \frac{1}{m_2}(k_2+k_3) & -\frac{1}{\sqrt{m_2 m_3}}k_3 \\ 0 & -\frac{1}{\sqrt{m_2 m_3}}k_3 & \frac{1}{m_3}k_3 \end{bmatrix} \therefore \text{symmetric} \\ \text{+ standard form.}$$

This procedure is valid for M a full symmetric matrix such as those encountered in the finite element formulation.

7-1



Disregard
 $u_2 = u_7 = 0$

Constraints are

$$u_1 = u_4$$

$$u_5 = u_8$$

There are 7 coordinates
 and 2 constraint eqs.

$$\therefore \text{DOF} = 5$$

7-2

7 Coordinates are $u_1, u_3, u_4, u_5, u_6, u_8, u_9$

4 of the above coordinates are associated with
 the two constraint equations, u_1, u_4, u_5, u_8 .

$$\begin{bmatrix} 1 & -1 & 0 & 0 \\ 0 & 0 & 1 & -1 \end{bmatrix} \begin{Bmatrix} u_1 \\ u_4 \\ u_5 \\ u_8 \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \end{Bmatrix}$$

Above can be written as
 where u_4 & u_8 are chosen
 as gen. coords.

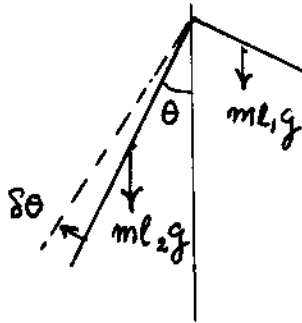
$$\begin{Bmatrix} u_1 \\ u_5 \end{Bmatrix} = \begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix} \begin{Bmatrix} u_4 \\ u_8 \end{Bmatrix}$$

$$\therefore \begin{Bmatrix} u_1 \\ u_3 \\ u_4 \\ u_5 \\ u_6 \\ u_8 \\ u_9 \end{Bmatrix} = \begin{bmatrix} 0 & 1 & 0 & 0 & 0 \\ 1 & 0 & 0 & 0 & 0 \\ 0 & 1 & 0 & 0 & 0 \\ 0 & 0 & 0 & 1 & 0 \\ 0 & 0 & 1 & 0 & 0 \\ 0 & 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 0 & 1 \end{bmatrix} \begin{Bmatrix} u_3 \\ u_4 \\ u_6 \\ u_8 \\ u_9 \end{Bmatrix}$$

gen. coords.

all coords.

7-3



$$-ml_2g \frac{1}{2}l_2[\cos\theta - \cos(\theta + \delta\theta)] + ml_1g \frac{l_1}{2}[\sin(\theta + \delta\theta) - \sin\theta] = 0$$

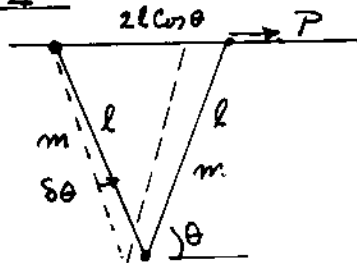
$$l_2^2[\cos\theta - (\cos\theta \cos\delta\theta - \sin\theta \sin\delta\theta)] + l_1^2[\sin\theta \cos\delta\theta + \cos\theta \sin\delta\theta] = 0$$

$$\cos\delta\theta \approx 1, \quad \sin\delta\theta \approx \delta\theta$$

$$l_2^2[\sin\theta \cdot \delta\theta] + l_1^2[\cos\theta \cdot \delta\theta] = 0$$

$$\therefore \tan\theta = \left(\frac{l_1}{l_2}\right)^2$$

7-4



$$U = 2mg \frac{l}{2} \sin\theta$$

$$\delta U = mgl \cos\theta \delta\theta$$

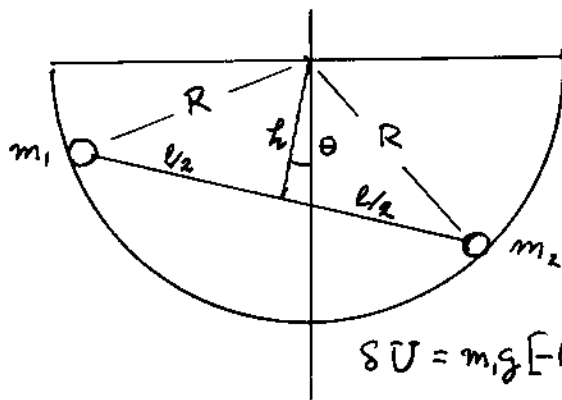
$$x = 2l \cos\theta$$

$$\delta x = -2l \sin\theta \delta\theta$$

$$\delta W = mgl \cos\theta \delta\theta - P 2l \sin\theta \delta\theta = 0$$

$$\therefore \tan\theta = \frac{mg}{2P}$$

7-5



$$h = \sqrt{R^2 - \left(\frac{l}{2}\right)^2}$$

$$2h = \sqrt{(2R)^2 - l^2}$$

$$U = m_1g \left[h \cos\theta - \frac{l}{2} \sin\theta \right] + m_2g \left[h \cos\theta + \frac{l}{2} \sin\theta \right]$$

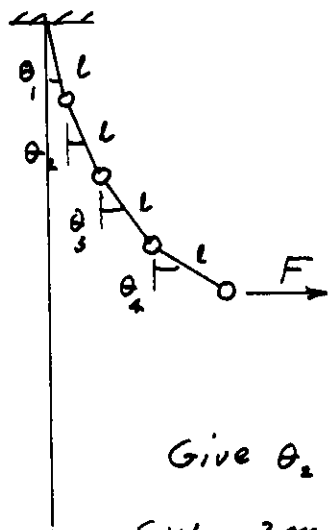
$$\delta U = m_1g \left[-h \sin\theta \delta\theta - \frac{l}{2} \cos\theta \delta\theta \right]$$

$$+ m_2g \left[-h \sin\theta \delta\theta + \frac{l}{2} \cos\theta \delta\theta \right] = 0$$

$$-(m_1 + m_2)h \sin\theta + (m_2 - m_1) \frac{l}{2} \cos\theta = 0$$

$$\tan\theta = \frac{(m_2 - m_1) \frac{l}{2}}{(m_1 + m_2) h} = \left(\frac{m_2 - m_1}{m_2 + m_1} \right) \frac{l}{\sqrt{(2R)^2 - l^2}}$$

7-6



Top mass position $x = l \sin \theta_1$
 $y = l \cos \theta_1$

Give θ_1 virtual displ $\delta \theta_1$

$$\delta x = l \cos \theta_1 \delta \theta_1$$

$$\delta y = -l \sin \theta_1 \delta \theta_1$$

$$\delta W = 4mg(-l \sin \theta_1 \delta \theta_1) + Fl \cos \theta_1 \delta \theta_1 = 0$$

$$\therefore \tan \theta_1 = \frac{F}{4mg}$$

Give θ_2 virtual displ. $\delta \theta_2$

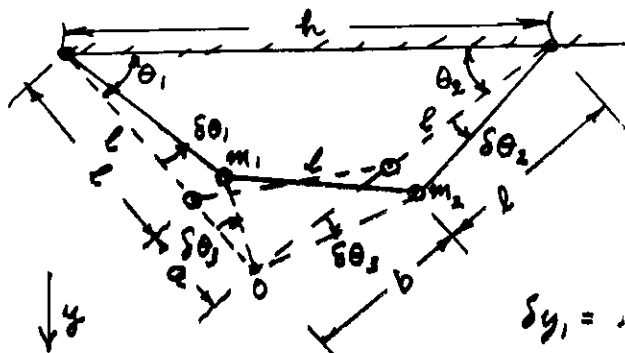
$$\delta W = 3mg(-l \sin \theta_2 \delta \theta_2) + Fl \cos \theta_2 \delta \theta_2 = 0$$

$$\therefore \tan \theta_2 = \frac{F}{3mg}$$

etc. $\tan \theta_3 = \frac{F}{2mg}, \quad \tan \theta_4 = \frac{F}{mg}$

7-8

dotted lines show original position in equilibrium



$$\delta W = m_1 g \delta y_1 - m_2 g \delta y_2 = 0$$

The middle length rotates $\delta \theta_3$ about O.

$$l \delta \theta_1 = a \delta \theta_3$$

$$l \delta \theta_2 = b \delta \theta_3$$

$$\delta y_1 = l \delta \theta_1 \cos \theta_1 = a \delta \theta_3 \cos \theta_1$$

$$\delta y_2 = l \delta \theta_2 \cos \theta_2 = b \delta \theta_3 \cos \theta_2$$

$$\therefore \delta W = (m_1 g a \cos \theta_1 - m_2 g b \cos \theta_2) \delta \theta_3 = 0$$

$$\therefore m_1 a \cos \theta_1 - m_2 b \cos \theta_2 = 0$$

Geometric Eqs.

$$(l+a) \cos \theta_1 + (l+b) \cos \theta_2 = h$$

$$l \sin \theta_1 - l \sin \theta_2 = b \sin \theta_2 - a \sin \theta_1$$

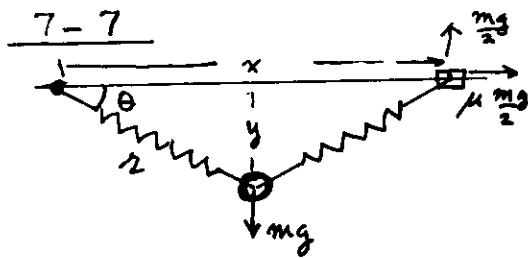
$$l^2 = (a \cos \theta_1 + b \cos \theta_2)^2 + (b \sin \theta_2 - a \sin \theta_1)^2$$

4 eqs

4 unknowns

a, b, θ_1, θ_2

$$\sin(\theta_1 + \theta_2) = \frac{h}{l} \frac{(m_1 \cos \theta_1 \sin \theta_2 - m_2 \cos \theta_2 \sin \theta_1)}{(m_1 \cos \theta_1 - m_2 \cos \theta_2)}$$



$$y = r \sin \theta$$

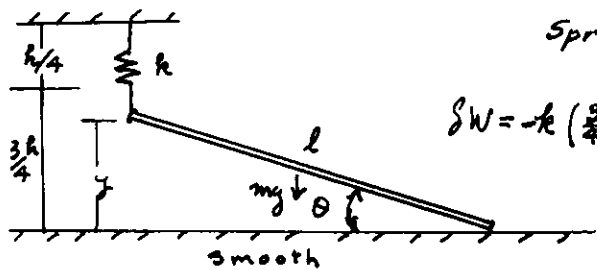
$$\delta y = r \cos \theta \delta \theta$$

$$x = 2r \cos \theta$$

$$\delta x = -2r \sin \theta \delta \theta$$

$$\delta W = mg \delta y + \mu \frac{mg}{2} \delta x = 0 \quad \therefore \tan \theta = \frac{1}{\mu}$$

7-9



$$\text{Stretch of spring} = \frac{3}{4}h - y$$

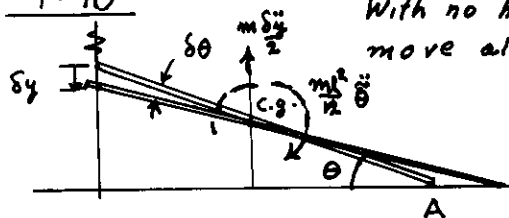
$$\text{Spring force} = k(\frac{3}{4}h - y)$$

$$\delta W = -k(\frac{3}{4}h - y)\delta y + mg \frac{\delta y}{2} = 0$$

$$\therefore y = \frac{3}{4}h - \frac{mg}{2k}$$

$$\sin \theta = \frac{y}{l} = \frac{1}{l} \left(\frac{3h}{4} - \frac{mg}{2k} \right)$$

7-10



With no horizontal force on the bar, the c.g. must move along the vertical line.

$$\text{Let } \delta \theta = \tilde{\theta} \quad \therefore \theta = \theta_0 - \tilde{\theta}$$

$$\sin(\theta_0 - \tilde{\theta}) \approx \sin \theta_0 - \tilde{\theta} \cos \theta_0$$

$$\cos(\theta_0 - \tilde{\theta}) \approx \cos \theta_0 + \tilde{\theta} \sin \theta_0$$

$$\text{Forces are balanced at } \theta_0. \quad \delta y = l \sin \theta_0 - l \sin(\theta_0 - \tilde{\theta}) = (l \cos \theta_0) \tilde{\theta}$$

$$\text{Spring force due to } \tilde{\theta} \text{ is } -k \delta y = -(k l \cos \theta_0) \tilde{\theta}$$

$$\delta W = (-k l \cos \theta_0) \tilde{\theta} \cdot (l \cos \theta_0 \tilde{\theta}) - \left(\frac{m}{2} l \cos \theta_0 \ddot{\tilde{\theta}} \right) \left(\frac{l \cos \theta_0}{2} \tilde{\theta} \right) - \left(\frac{m l^2}{12} \ddot{\tilde{\theta}} \right) \tilde{\theta} = 0$$

$$\left(\frac{m l^2}{12} + \frac{m l^2 \cos^2 \theta_0}{4} \right) \ddot{\tilde{\theta}} + (k l^2 \cos^2 \theta_0) \tilde{\theta} = 0$$

For small θ_0 above eq. reduces to

$$\left(\frac{m l^2}{3} \right) \ddot{\tilde{\theta}} + (k l^2) \tilde{\theta} = 0$$

7-11

$$\left[(m l_1) \frac{l_1^2}{3} + (m l_2) \frac{l_2^2}{3} \right] \ddot{\theta} = -(m l_2 g) \frac{l_2}{2} \sin \theta + (m l_1 g) \frac{l_1}{2} \cos \theta$$

$$\text{Let } \theta = \theta_0 + \theta_n$$

$$\sin \theta \cong \sin \theta_0 + \theta_n \cos \theta_0$$

$$\cos \theta \cong \cos \theta_0 - \theta_n \sin \theta_0$$

$$\text{but } -(m l_2 g) \frac{l_2}{2} \sin \theta_0 + (m l_1 g) \frac{l_1}{2} \cos \theta_0 = 0$$

$$\therefore \left(\frac{m l_1^3}{3} + \frac{m l_2^3}{3} \right) \ddot{\theta}_n = -\frac{m g}{2} (l_2^2 \cos \theta_0 + l_1^2 \sin \theta_0) \theta_n$$

$$\ddot{\theta}_n + \frac{3}{2} g \left(\frac{l_2^2 \cos \theta_0 + l_1^2 \sin \theta_0}{l_1^3 + l_2^3} \right) \theta_n = 0$$

$$\text{where } \tan \theta_0 = \left(\frac{l_1}{l_2} \right)^2$$

7-12

$$T = \frac{1}{2} (m_1 + m_2) R^2 \dot{\theta}^2$$

$$\frac{d}{dt} \left(\frac{\partial T}{\partial \dot{\theta}} \right) = (m_1 + m_2) R^2 \ddot{\theta}$$

$$h = \sqrt{R^2 + \left(\frac{R}{2} \right)^2}$$

$$U = -(m_1 + m_2) g h \cos \theta - (m_2 - m_1) g \frac{R}{2} \sin \theta$$

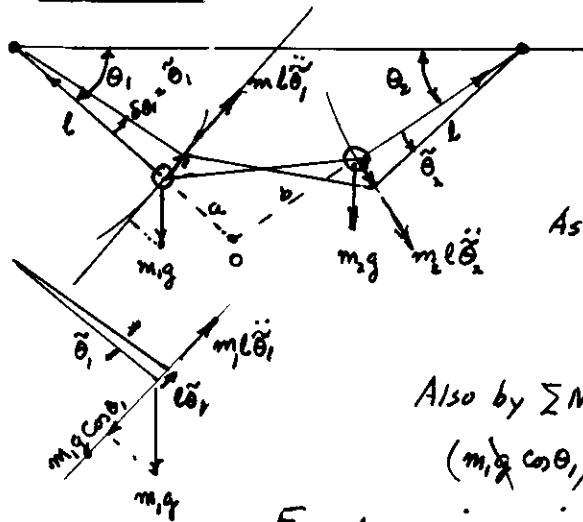
$$\frac{\partial U}{\partial \theta} = (m_1 + m_2) g h \sin \theta - (m_2 - m_1) g \frac{R}{2} \cos \theta$$

Lagrange's Eq.

$$(m_1 + m_2) R^2 \ddot{\theta} + (m_1 + m_2) g h \sin \theta - (m_2 - m_1) g \frac{R}{2} \cos \theta = 0$$

$$\text{Let } \theta = \theta_0 + \theta_n$$

$$(m_1 + m_2) R^2 \ddot{\theta}_n + \left[(m_1 + m_2) g h \cos \theta_0 + (m_2 - m_1) g \frac{R}{2} \sin \theta_0 \right] \theta_n = 0$$



Motion of each mass is along circle of radius l . Virtual displ. $= l\tilde{\theta}_1$ and $l\tilde{\theta}_2$

where $\theta_1 = \bar{\theta}_1 + \tilde{\theta}_1$ and $\theta_2 = \bar{\theta}_2 + \tilde{\theta}_2$

As in Prob 7-8

$$s\theta_1 = \tilde{\theta}_1 = \frac{a}{l} s\theta_2 = \frac{a}{l} \tilde{\theta}_2$$

$$s\theta_2 = \tilde{\theta}_2 = \frac{b}{l} s\theta_2 = \frac{b}{l} \tilde{\theta}_2 = \frac{b}{a} \tilde{\theta}_1$$

Also by ΣM_O

$$(m_1 g \cos \theta_1) a = (m_2 g \cos \theta_2) b \quad \text{same eq. as Prob 7-8}$$

For dynamics include inertia forces $m_1 l \ddot{\theta}_1$ & $m_2 l \ddot{\theta}_2$ in tangential direction $l\tilde{\theta}_1$ and $l\tilde{\theta}_2$. Then work done by virtual displ. $l\tilde{\theta}_1$ and $l\tilde{\theta}_2$ is

$$\delta W = (m_1 l \ddot{\theta}_1 - m_1 g \cos \theta_1) l \tilde{\theta}_1 + (m_2 l \ddot{\theta}_2 + m_2 g \cos \theta_2) l \tilde{\theta}_2 = 0$$

For small $\tilde{\theta}$ with $\theta = \bar{\theta} \pm \tilde{\theta}$

$$\cos \theta_1 = \cos \bar{\theta}_1 - \tilde{\theta}_1 \sin \bar{\theta}_1$$

$$\cos \theta_2 = \cos \bar{\theta}_2 + \tilde{\theta}_2 \sin \bar{\theta}_2$$

$$\tilde{\theta}_2 = \frac{b}{a} \tilde{\theta}_1 = \frac{m_1}{m_2} \frac{\cos \bar{\theta}_1}{\cos \bar{\theta}_2} \tilde{\theta}_1$$

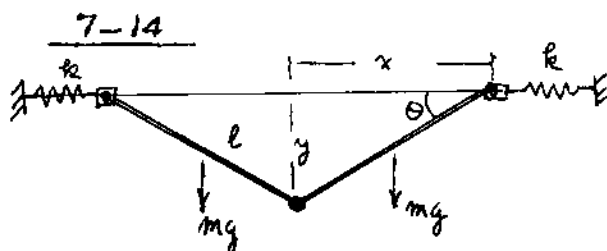
$$+ [m_1 l + m_2 l \left(\frac{b}{a}\right)^2] \ddot{\theta}_1 \tilde{\theta}_1 - g \left\{ m_1 (\cos \bar{\theta}_1 - \tilde{\theta}_1 \sin \bar{\theta}_1) \tilde{\theta}_1 - m_2 \left(\cos \bar{\theta}_2 + \frac{b}{a} \tilde{\theta}_1 \sin \bar{\theta}_2 \right) \frac{b}{a} \tilde{\theta}_1 \right\} = 0$$

Since $m_1 \cos \bar{\theta}_1 \cdot a - m_2 \cos \bar{\theta}_2 \cdot b = 0$ from Prob. 8-6 & above

$$\left[m_1 l + m_2 l \left(\frac{b}{a}\right)^2 \right] \ddot{\theta}_1 \tilde{\theta}_1 + g \left\{ m_1 \sin \bar{\theta}_1 + m_2 \left(\frac{m_1 \cos \bar{\theta}_1}{m_2 \cos \bar{\theta}_2} \right)^2 \right\} \tilde{\theta}_1 \tilde{\theta}_1 = 0$$

$$\left[m_1 + m_2 \left(\frac{m_1 \cos \bar{\theta}_1}{m_2 \cos \bar{\theta}_2} \right)^2 \right] \ddot{\theta}_1 + \frac{g}{l} \left[m_1 \sin \bar{\theta}_1 + m_2 \left(\frac{m_1 \cos \bar{\theta}_1}{m_2 \cos \bar{\theta}_2} \right)^2 \right] \tilde{\theta}_1 = 0$$

$$\therefore \omega_n^2 = \frac{g}{l} \cdot \frac{m_1 \sin \bar{\theta}_1 + m_2 \left(\frac{m_1 \cos \bar{\theta}_1}{m_2 \cos \bar{\theta}_2} \right)^2}{m_1 + m_2 \left(\frac{m_1 \cos \bar{\theta}_1}{m_2 \cos \bar{\theta}_2} \right)^2}$$



Let $\theta_0 = \text{equilib angle}$

Spring force = F_{s0} at $\theta = \theta_0$

$$x = l \cos \theta \quad \delta x = -l \sin \theta_0 \delta \theta$$

$$y = l \sin \theta \quad \delta y = l \cos \theta_0 \delta \theta$$

$$\delta W = 2mg \frac{\delta y}{2} + 2F_{s0} \delta x = 0$$

$$(mg \cos \theta_0 - 2F_{s0} \sin \theta_0) l \delta \theta = 0$$

If $F_s = 0$ at $\theta = 0$ then $F_{s0} = k(l - x) = k l (1 - \cos \theta_0)$

$$\tan \theta_0 = \frac{mg}{2kl(1 - \cos \theta_0)}$$

solve by trial for given
value of $\frac{mg}{2kl}$

7-15

$$T = \frac{1}{2} m_0 [\dot{r}^2 + r^2 \dot{\theta}^2] + \frac{1}{2} \left(m \frac{l^2}{3} \right) \dot{\theta}^2$$

$$U = \frac{1}{2} k (r - r_0)^2 - m_0 g r \cos \theta - mg \frac{l}{2} \cos \theta$$

$$\frac{\partial T}{\partial \dot{\theta}} = m_0 r^2 \dot{\theta} + m \frac{l^2}{3} \dot{\theta} \quad \frac{\partial T}{\partial \theta} = 0$$

$$\frac{\partial U}{\partial \theta} = m_0 g r \sin \theta + mg \frac{l}{2} \sin \theta$$

$$\frac{d}{dt} \left(\frac{\partial T}{\partial \dot{\theta}} \right) - \frac{\partial T}{\partial \theta} + \frac{\partial U}{\partial \theta} = 0$$

$$m_0 [r \ddot{\theta} + 2 \dot{r} \dot{\theta}] r + \frac{m l^2}{3} \ddot{\theta} + (m_0 g r + mg \frac{l}{2}) \sin \theta = 0$$

$$\frac{\partial T}{\partial \dot{r}} = m_0 \dot{r}$$

$$\frac{\partial T}{\partial r} = m_0 r \dot{\theta}^2$$

$$\frac{\partial U}{\partial r} = -m_0 g \cos \theta + k(r - r_0)$$

$$\frac{d}{dt} \left(\frac{\partial T}{\partial \dot{r}} \right) - \frac{\partial T}{\partial r} + \frac{\partial U}{\partial r} = 0$$

$$m_0 \ddot{r} - m_0 r \dot{\theta}^2 + k(r - r_0) - m_0 g \cos \theta = 0$$

-7-16

$$T = \frac{1}{2} \int_0^l m \dot{y}^2 dx$$

$$U = \frac{1}{2} k y^2(l) + \frac{1}{2} K y'^2(0) + \frac{1}{2} \int_0^l EI \left(\frac{d^2 y}{dx^2} \right)^2 dx$$

$$y = \frac{x}{l} q_1 + q_2 \sin \frac{\pi x}{l}$$

$$\dot{y}^2 = \left(\frac{x}{l} \right)^2 \dot{q}_1^2 + 2 \left(\frac{x}{l} \right) \dot{q}_1 \dot{q}_2 \sin \frac{\pi x}{l} + \dot{q}_2^2 \sin^2 \frac{\pi x}{l}$$

$$y' = \frac{1}{l} q_1 + q_2 \frac{\pi}{l} \cos \frac{\pi x}{l}$$

$$y'^2 = \frac{1}{l^2} q_1^2 + \frac{2}{l} q_1 q_2 \frac{\pi}{l} \cos \frac{\pi x}{l} + q_2^2 \left(\frac{\pi}{l} \right)^2 \cos^2 \frac{\pi x}{l}$$

$$y'' = -q_2 \left(\frac{\pi}{l} \right)^2 \sin \frac{\pi x}{l}$$

$$\frac{d}{dt} \frac{\partial T}{\partial \dot{q}_1} = \ddot{q}_1 \int_0^l m \left(\frac{x}{l} \right)^2 dx + \ddot{q}_2 \int_0^l \frac{x}{l} \sin \frac{\pi x}{l} dx = \frac{ml}{3} \ddot{q}_1 + \frac{l}{\pi} \ddot{q}_2$$

$$\frac{d}{dt} \frac{\partial T}{\partial \dot{q}_2} = \ddot{q}_1 \int_0^l \frac{x}{l} \sin \frac{\pi x}{l} dx + \ddot{q}_2 \int_0^l \sin^2 \frac{\pi x}{l} dx = \frac{l}{\pi} \ddot{q}_1 + \frac{l}{2} \ddot{q}_2$$

$$\frac{\partial U}{\partial q_1} = k q_1 + \frac{K}{l^2} q_1 + \frac{K \pi}{l^2} q_2$$

$$\frac{\partial U}{\partial q_2} = \frac{K \pi}{l^2} q_1 + K \left(\frac{\pi}{l} \right)^2 q_2 + EI \left(\frac{\pi}{l} \right)^4 \frac{l}{2} q_2$$

$$ml \begin{bmatrix} \frac{1}{3} & \frac{1}{2\pi} \\ \frac{1}{2\pi} & \frac{1}{2} \end{bmatrix} \begin{Bmatrix} \ddot{q}_1 \\ \ddot{q}_2 \end{Bmatrix} + \frac{EI}{l^3} \begin{bmatrix} (k + \frac{K}{l^2}) \frac{l^3}{EI} & -\frac{\pi K}{l^2} \frac{l^3}{EI} \\ -\frac{\pi K}{l^2} \frac{l^3}{EI} & (\frac{\pi^4}{2} + \frac{\pi^2 K}{l^2} \frac{l^3}{EI}) \end{bmatrix} \begin{Bmatrix} q_1 \\ q_2 \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \end{Bmatrix}$$

$$T = \frac{1}{2} \left(\frac{m\ell^2}{3} \right) \dot{\theta}_1^2 + \frac{1}{2} \left(\frac{m\ell^2}{12} \right) \dot{\theta}_2^2 + \frac{1}{2} m (\ell \dot{\theta}_1 + \frac{\ell}{2} \dot{\theta}_2)^2$$

$$U = \frac{1}{2} k \left(\frac{\ell}{2} \theta_1 \right)^2 + mg \frac{\ell}{2} (1 - \cos \theta_1) + mg \left[\ell (1 - \cos \theta_1) + \frac{\ell}{2} (1 - \cos \theta_2) \right]$$

$$\frac{d}{dt} \frac{\partial T}{\partial \dot{\theta}_1} = \left(\frac{m\ell^2}{3} \right) \ddot{\theta}_1 + m \left(\ell \ddot{\theta}_1 + \frac{\ell}{2} \ddot{\theta}_2 \right) \ell$$

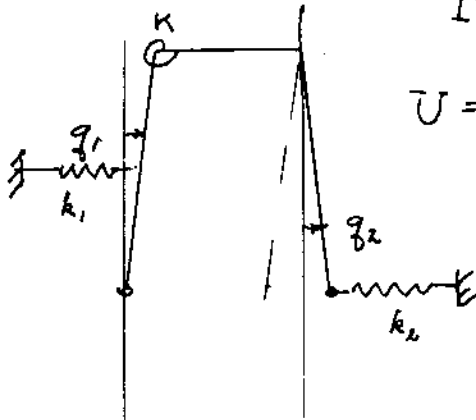
$$\frac{d}{dt} \frac{\partial T}{\partial \dot{\theta}_2} = \left(\frac{m\ell^2}{12} \right) \ddot{\theta}_2 + m \left(\ell \ddot{\theta}_1 + \frac{\ell}{2} \ddot{\theta}_2 \right) \frac{\ell}{2}$$

$$\frac{\partial U}{\partial \theta_1} = k \left(\frac{\ell}{2} \theta_1 \right) + mg \frac{\ell}{2} \sin \theta_1 + mg \ell \sin \theta_1$$

$$\frac{\partial U}{\partial \theta_2} = mg \frac{\ell}{2} \sin \theta_2$$

$$\begin{bmatrix} \left(\frac{m\ell^2}{3} + m\ell^2 \right) & m\frac{\ell^2}{2} \\ m\frac{\ell^2}{2} & \left(\frac{m\ell^2}{12} + \frac{m\ell^2}{4} \right) \end{bmatrix} \begin{Bmatrix} \ddot{\theta}_1 \\ \ddot{\theta}_2 \end{Bmatrix} + \begin{bmatrix} \left(k\frac{\ell}{2} + \frac{3}{2}mg\ell \right) & 0 \\ 0 & mg\frac{\ell}{2} \end{bmatrix} \begin{Bmatrix} \theta_1 \\ \theta_2 \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \end{Bmatrix}$$

7-18



$$T = \frac{1}{2}(m_1 + m_2)(2l\dot{q}_1)^2 + \frac{1}{2}J_1\dot{q}_1^2 + \frac{1}{2}J_2\dot{q}_2^2$$

$$U = \frac{1}{2}k_1(lq_1)^2 + \frac{1}{2}Kq_1^2 + \frac{1}{2}k_2(2l)^2(q_1 + q_2)^2$$

$$\frac{d}{dt} \frac{\partial T}{\partial \dot{q}_1} = (m_1 + m_2)(4l^2)\ddot{q}_1 + J_1\ddot{q}_1$$

$$\frac{d}{dt} \frac{\partial T}{\partial \dot{q}_2} = J_2\ddot{q}_2$$

$$\frac{\partial U}{\partial q_1} = l^2 k_1 q_1 + K q_1 + k_2 (4l^2)(q_1 + q_2)$$

$$\frac{\partial U}{\partial q_2} = 4l^2 k_2 (q_1 + q_2)$$

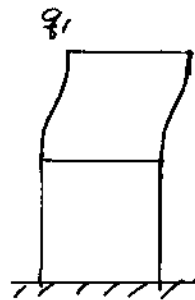
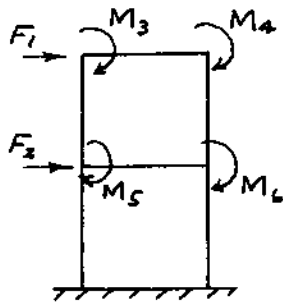
$$[(m_1 + m_2)4l^2 + J_1]\ddot{q}_1 + [l^2 k_1 + K + 4l^2 k_2]q_1 + 4l^2 k_2 q_2 = 0$$

$$J_2\ddot{q}_2 + 4l^2 k_2 (q_1 + q_2) = 0$$

7-19

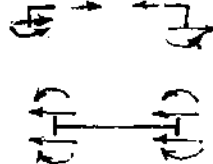
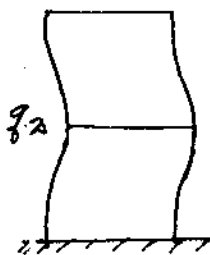
Refer to FIG 7.1-4.

Assume $l_2 = l_1 = l$

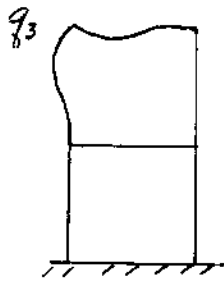


$$\begin{Bmatrix} F_1 \\ F_2 \\ M_3 \\ M_4 \\ M_5 \\ M_6 \end{Bmatrix} = \frac{EI}{l^3} \begin{bmatrix} 24 & 0 & 0 & 0 & 0 & 0 \\ -24 & & & & & \\ -6l & & & & & \\ -6l & & & & & \\ -6l & & & & & \\ -6l & & & & & \end{bmatrix} \begin{Bmatrix} q_1 \\ 0 \\ 0 \\ 0 \\ 0 \\ 0 \end{Bmatrix}$$

Examine FBD of each of the four corners for above
Refer to table 7.3-1

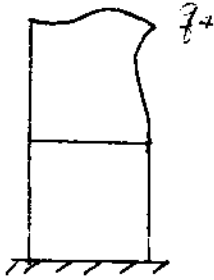


$$\begin{Bmatrix} F_1 \\ F_2 \\ M_3 \\ M_4 \\ M_5 \\ M_6 \end{Bmatrix} = \frac{EI}{l^3} \begin{bmatrix} 0 & -24 & 0 & 0 & 0 & 0 \\ & 48 & & & & \\ & 6l & & & & \\ & 6l & & & & \\ & 0 & & & & \\ & 0 & & & & \end{bmatrix} \begin{Bmatrix} 0 \\ q_2 \\ 0 \\ 0 \\ 0 \\ 0 \end{Bmatrix}$$

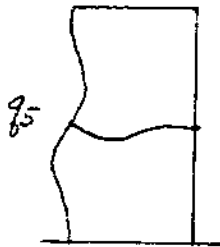


Examine FBD
of each corner

$$\begin{Bmatrix} F_1 \\ F_2 \\ M_3 \\ M_4 \\ M_5 \\ M_6 \end{Bmatrix} = \frac{EI}{l^3} \begin{bmatrix} 0 & 0 & -6l & 0 & 0 & 0 \\ 0 & 0 & 6l & 0 & 0 & 0 \\ 0 & 0 & 8l^2 & 2l^2 & 2l^2 & 0 \\ 0 & 0 & 2l^2 & 8l^2 & 2l^2 & 0 \\ 0 & 0 & 2l^2 & 2l^2 & 8l^2 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 \end{bmatrix} \begin{Bmatrix} 0 \\ 0 \\ q_3 \\ 0 \\ 0 \\ 0 \end{Bmatrix}$$



$$\begin{Bmatrix} F_1 \\ F_2 \\ M_3 \\ M_4 \\ M_5 \\ M_6 \end{Bmatrix} = \frac{EI}{l^3} \begin{bmatrix} 0 & 0 & 0 & -6l & 0 & 0 \\ 0 & 0 & 0 & 6l & 0 & 0 \\ 0 & 0 & 0 & 2l^2 & 0 & 0 \\ 0 & 0 & 0 & 8l^2 & 0 & 0 \\ 0 & 0 & 0 & 0 & 12l^2 & 0 \\ 0 & 0 & 0 & 2l^2 & 0 & 0 \end{bmatrix} \begin{Bmatrix} 0 \\ 0 \\ 0 \\ q_4 \\ 0 \\ 0 \end{Bmatrix}$$



$$\begin{Bmatrix} F_1 \\ F_2 \\ M_3 \\ M_4 \\ M_5 \\ M_6 \end{Bmatrix} = \frac{EI}{l^3} \begin{bmatrix} 0 & 0 & 0 & 0 & -6l & 0 \\ 0 & 0 & 0 & 0 & 6l & 0 \\ 0 & 0 & 0 & 0 & 2l^2 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 12l^2 & 0 \end{bmatrix} \begin{Bmatrix} 0 \\ 0 \\ 0 \\ 0 \\ q_5 \\ 0 \end{Bmatrix}$$



$$\begin{Bmatrix} F_1 \\ F_2 \\ M_3 \\ M_4 \\ M_5 \\ M_6 \end{Bmatrix} = \frac{EI}{l^3} \begin{bmatrix} 0 & 0 & 0 & 0 & 0 & -6l \\ 0 & 0 & 0 & 0 & 0 & 6l \\ 0 & 0 & 0 & 0 & 0 & 2l^2 \\ 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 2l^2 \\ 0 & 0 & 0 & 0 & 0 & 12l^2 \end{bmatrix} \begin{Bmatrix} 0 \\ 0 \\ 0 \\ 0 \\ 0 \\ q_6 \end{Bmatrix}$$

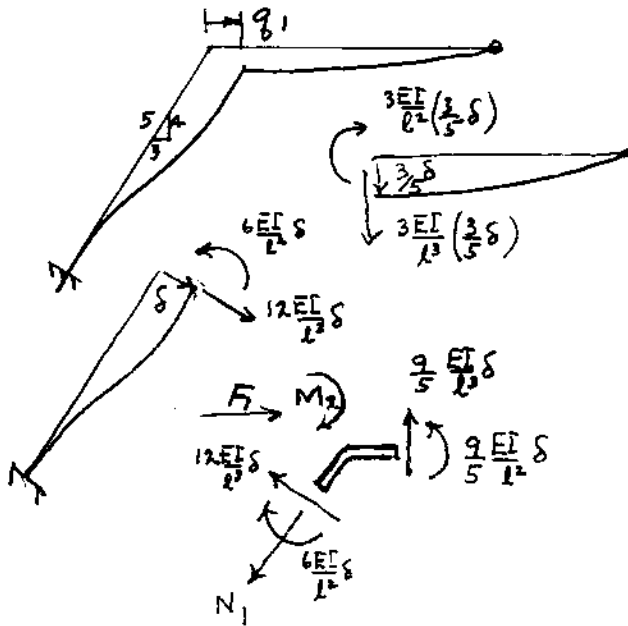
$$[k] = \frac{EI}{l^3} \begin{bmatrix} 24 & -24 & -6l & -6l & -6l & -6l \\ -24 & 48 & 6l & 6l & 0 & 0 \\ -6l & 6l & 8l^2 & 2l^2 & 2l^2 & 0 \\ -6l & 6l & 2l^2 & 8l^2 & 0 & 2l^2 \\ -6l & 0 & 2l^2 & 0 & 12l^2 & 2l^2 \\ -6l & 0 & 0 & 2l^2 & 2l^2 & 12l^2 \end{bmatrix}$$

7-19 Cont.

$$[m] = \begin{bmatrix} 2m & 0 & 0 & 0 & 0 & 0 \\ 0 & 2m & 0 & 0 & 0 & 0 \\ 0 & 0 & J & 0 & 0 & 0 \\ 0 & 0 & 0 & J & 0 & 0 \\ 0 & 0 & 0 & 0 & J & 0 \\ 0 & 0 & 0 & 0 & 0 & J \end{bmatrix}$$

$$[m]\{\ddot{q}\} + [k]\{q\} = 0$$

7-20



$$\sum F_y = -\frac{4}{5}N_1 + \left(\frac{9}{5} + \frac{3}{5} \times 12\right) \frac{EI}{l^3} \delta = 0$$

$$\therefore N_1 = \frac{45}{4} \frac{EI}{l^3} \delta$$

$$\sum F_x = -F + \left[\frac{3}{5} \left(\frac{45}{4}\right) + \frac{4}{5} \times 12\right] \frac{EI}{l^3} \delta = 0$$

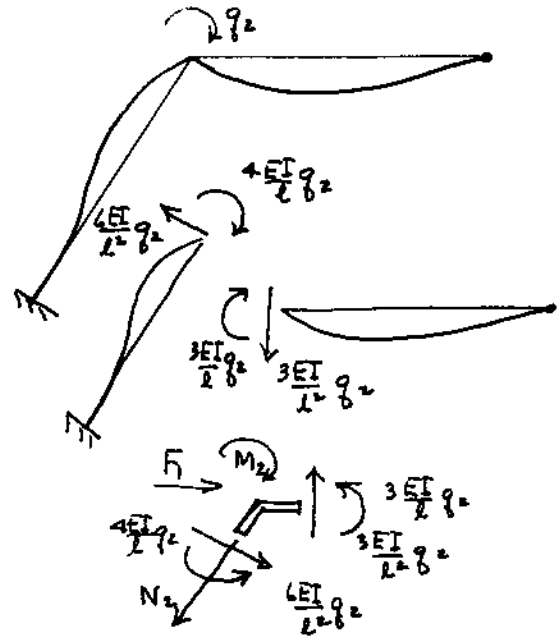
$$F_1 = 16.35 \frac{EI}{l^3} \delta$$

$$\sum M = M_2 = \left(\frac{9}{5} - 6\right) \frac{EI}{l^2} \delta = -4.20 \frac{EI}{l^2} \delta$$

$$\text{but } \delta = \frac{5}{4} q_1$$

$$\therefore F_1 = 20.44 \frac{EI}{l^3} q_1$$

$$M_2 = -5.25 \frac{EI}{l^2} q_1$$



$$\sum F_y = 3 \frac{EI}{l^2} q_2 - \frac{3}{5} \left(6 \frac{EI}{l^3}\right) q_2 - \frac{4}{5} N_2 = 0$$

$$\therefore N_2 = -0.75 \frac{EI}{l^2} q_2$$

$$\sum F_x = 0 \text{ gives}$$

$$F_1 = \left[\frac{3}{5}(-0.75) - 6 \times \frac{4}{5}\right] \frac{EI}{l^2} q_2$$

$$F_1 = -5.25 \frac{EI}{l^2} q_2$$

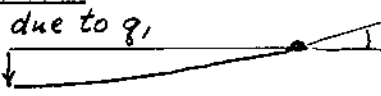
$$M_2 = 7 \frac{EI}{l} q_2$$

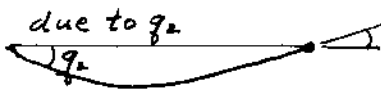
7-20 Cont.

$$\begin{Bmatrix} F_1 \\ M_2 \end{Bmatrix} = \frac{EI}{l^3} \begin{bmatrix} 20.44 & -5.25l \\ -5.25l & 7.0l^2 \end{bmatrix} \begin{Bmatrix} q_1 \\ q_2 \end{Bmatrix}$$

7-21

From Table 6.3-1 rotation of right end is

due to q_1 , $\frac{3}{5}\delta$  $\frac{3}{2} \left(\frac{3}{5} \frac{\delta}{l} \right) = \frac{9}{8} \frac{\delta}{l}$

due to q_2  $\frac{1}{2} q_2$

$$T = \frac{1}{2} (m_1 + m_2) \dot{q}_1^2 + \frac{1}{2} J_1 \dot{q}_2^2 + \frac{1}{2} J_2 \left(\frac{9}{8} \frac{\dot{q}_1}{l} + \frac{1}{2} \dot{q}_2 \right)^2$$

$$\frac{d}{dt} \frac{\partial T}{\partial \dot{q}_1} = (m_1 + m_2) \ddot{q}_1 + J_2 \left(\frac{9}{8l} \ddot{q}_1 + \frac{1}{2} \ddot{q}_2 \right) \frac{9}{8l}$$

$$\frac{d}{dt} \frac{\partial T}{\partial \dot{q}_2} = J_1 \ddot{q}_2 + J_2 \left(\frac{9}{8l} \dot{q}_1 + \frac{1}{2} \dot{q}_2 \right) \frac{1}{2}$$

$$U = \frac{1}{2} (q_1, q_2) \begin{bmatrix} k_0 \\ k_0 \end{bmatrix} \begin{Bmatrix} q_1 \\ q_2 \end{Bmatrix} + \frac{1}{2} k_0 q_1^2 + \frac{1}{2} K_0 \left(\frac{9}{8l} q_1 + \frac{1}{2} q_2 \right)^2$$

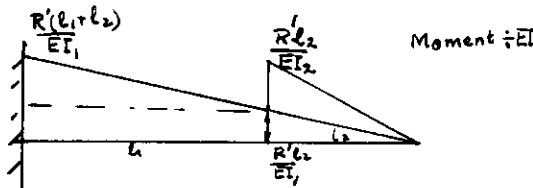
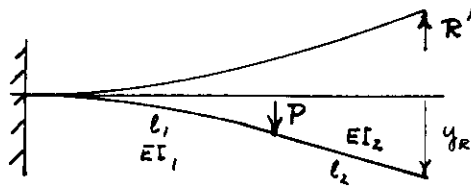
$$\frac{\partial U}{\partial q_{1,2}} = \begin{bmatrix} \left\{ 20.44 \frac{EI}{l^3} + k_0 + \left(\frac{9}{8l} \right)^2 K_0 \right\} & \left\{ -5.25 \frac{EI}{l^2} + \frac{1}{2} \times \frac{9}{8l} K_0 \right\} \\ \left\{ -5.25 \frac{EI}{l^2} + \frac{1}{2} \times \frac{9}{8l} K_0 \right\} & \left\{ 7.0 \frac{EI}{l} + \frac{1}{4} K_0 \right\} \end{bmatrix} \begin{Bmatrix} q_1 \\ q_2 \end{Bmatrix}$$

Eq. of motion

$$\begin{bmatrix} \left\{ m_1 + m_2 + \left(\frac{9}{8l} \right)^2 J_2 \right\} & \left\{ \frac{1}{2} \times \frac{9}{8l} J_2 \right\} \\ \left\{ \frac{1}{2} \times \frac{9}{8l} J_2 \right\} & \left\{ J_1 + \frac{1}{4} J_2 \right\} \end{bmatrix} \begin{Bmatrix} \ddot{q}_1 \\ \ddot{q}_2 \end{Bmatrix}$$

$$+ \begin{bmatrix} \left\{ 20.44 \frac{EI}{l^3} + k_0 + \left(\frac{9}{8l} \right)^2 K_0 \right\} & \left\{ -5.25 \frac{EI}{l^2} + \frac{9}{16l} K_0 \right\} \\ \left\{ -5.25 \frac{EI}{l^2} + \frac{9}{16l} K_0 \right\} & \left\{ 7.0 \frac{EI}{l} + \frac{1}{4} K_0 \right\} \end{bmatrix} \begin{Bmatrix} q_1 \\ q_2 \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \end{Bmatrix}$$

Assign numbers for $m_1, m_2, J_1, J_2, k_0, K_0$ & l
for normal mode determination.



Due to P

$$y_P = \frac{Pl_1^3}{3EI_1} \quad y_P' = \frac{Pl_1^2}{2EI_1}$$

$\therefore y_R = \frac{Pl_1^3}{3EI_1} + \left(\frac{Pl_1^2}{2EI_1}\right)l_2$

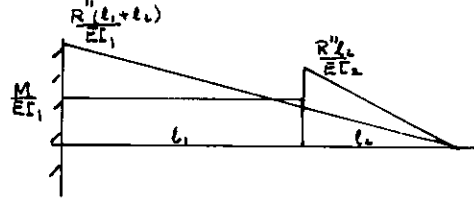
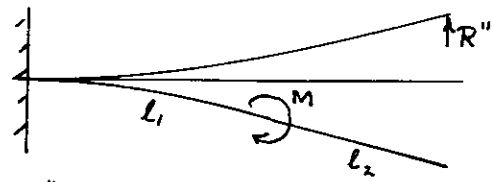
Due to R'

$$y_R = \left(\frac{1}{2} \frac{R'l_2}{EI_2} l_2\right) \frac{2}{3} l_2 + \left(\frac{R'l_2}{EI_2} \cdot l_1\right) \left(l_2 + \frac{1}{2} l_1\right) + \left(\frac{1}{2} \frac{R'l_1}{EI_1} \cdot l_1\right) \left(l_2 + \frac{2}{3} l_1\right)$$

Equate y_R due to P and due to R' (use $\frac{EI_1}{EI_2} = 2$)

$$\frac{P}{EI_1} \left[\frac{l_1^3}{3} + \frac{l_1^2 l_2}{2}\right] = \frac{R'}{EI_1} \left[\frac{2}{3} l_2^3 + l_1 l_2^2 + l_2 l_1^2 + \frac{l_1^3}{3}\right]$$

$$= R' [b]$$



Due to M

$$y_M = \left(\frac{M}{EI_1} \cdot l_1\right) \frac{l_1}{2}$$

$y_M' = \frac{M}{EI_1} l_1$

$y_R = \frac{M}{EI_1} \frac{l_1^2}{2} + \frac{M l_1 l_2}{EI_1}$

Due to R''

y_R same as that for R' with R' replaced by R''

Equate y_R due to M and due to R''

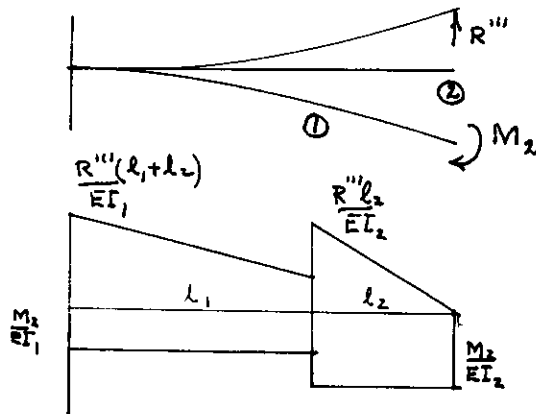
$$\frac{M}{EI_1} \left(\frac{l_1^2}{2} + l_1 l_2\right) = \frac{R''}{EI_1} \left[\frac{2}{3} l_2^3 + l_1 l_2^2 + l_2 l_1^2 + \frac{l_1^3}{3}\right]$$

$$= R'' [b]$$

$M_1 = -M - Pl_1 + R(l_1 + l_2) \quad \text{where } R = R' + R''$

$$R = \frac{P\left(\frac{1}{3} l_1^3 + \frac{1}{2} l_1^2 l_2\right) + M\left(\frac{1}{2} l_1^2 + l_1 l_2\right)}{\frac{2}{3} l_2^3 + l_1 l_2^2 + l_2 l_1^2 + \frac{1}{3} l_1^3}$$

Need stiffness matrix for M_1 , P and M_2 at end of beam



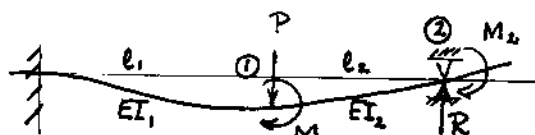
$$y_R = \frac{M_2}{EI_1} l_1 \left(l_2 + \frac{l_1}{2}\right) + \frac{M_2}{EI_2} \frac{l_2^2}{2}$$

$$y_R = \frac{R'''}{EI_1} \left[\frac{2}{3} l_2^3 + l_1 l_2^2 + l_2 l_1^2 + \frac{l_1^3}{3}\right]$$

Equate

$$\frac{M_2}{EI_1} \left[l_1 l_2 + \frac{l_1^2}{2} + 2 \cdot \frac{l_1^2}{2}\right] = \frac{R'''}{EI_1} [b]$$

7-23 Cont.



Determine y_1 , θ_1 , y_2 , θ_2

Set $y_2 = 0$ to find R

Determine flexibility matrix & invert.
 $EI_1 = 2 EI_2$

$$y_1 = \frac{Pl_1^3}{8EI_1} + \frac{Ml_1^2}{2EI_1} + \frac{M_2l_1^2}{2EI_1} - \frac{1}{2} \left[\frac{R(l_1+l_2)l_1}{EI_1} - \frac{Rl_1l_2}{EI_1} \right] \frac{2}{3}l_1 - \frac{Rl_2l_1^2}{EI_1 \cdot 2}$$

$$\theta_1 = \frac{Pl_1^2}{2EI_1} + \frac{Ml_1}{EI_1} + \frac{M_2l_1}{EI_1} - \frac{1}{2} \left[\frac{R(l_1+l_2)}{EI_1} + \frac{Rl_2}{EI_1} \right] l_1$$

$$y_2 = \frac{1}{2} \frac{Pl_1^2}{EI_1} \left(\frac{2}{3}l_1 + l_2 \right) + \frac{Ml_1}{EI_1} \left(\frac{1}{2}l_1 + l_2 \right) + \frac{M_2l_1}{EI_1} \left(\frac{l_1}{2} + l_2 \right) + \frac{M_2}{EI_2} \frac{l_2^2}{2} \\ - \frac{1}{2} \frac{R(l_1+l_2)}{EI_1} (l_1+l_2) \frac{2}{3}(l_1+l_2) - \frac{1}{2} \left[\frac{Rl_2^2}{EI_2} - \frac{Rl_2l_2}{EI_1} \right] \frac{2}{3}l_2$$

$$\theta_2 = \frac{Pl_1^2}{2EI_1} + \frac{Ml_1}{EI_1} + \frac{M_2l_1}{EI_1} + \frac{M_2l_2}{EI_2} - \frac{1}{2} \frac{R(l_1+l_2)^2}{EI_1} - \frac{1}{2} \left(\frac{Rl_2}{EI_2} - \frac{Rl_2}{EI_1} \right) l_2$$

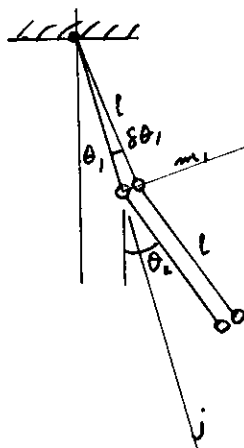
To simplify algebra let $l_1 = l_2 = l$ & $EI_1 = 2EI_2$, result is

$$\begin{Bmatrix} y_1 \\ \theta_1 \\ \theta_2 \\ y_2=0 \end{Bmatrix} = \frac{1}{EI_1} \begin{bmatrix} \frac{l^3}{3} & \frac{l^2}{2} & \frac{l^2}{2} & -\frac{5}{6}l^3 \\ \frac{l^2}{2} & l & l & -\frac{3}{2}l^2 \\ \frac{l^2}{2} & l & 3l & -\frac{5}{2}l^2 \\ \frac{5}{6}l^3 & \frac{3}{2}l^2 & \frac{5}{2}l^2 & -3l^3 \end{bmatrix} \begin{Bmatrix} P \\ M \\ M_2 \\ R \end{Bmatrix}$$

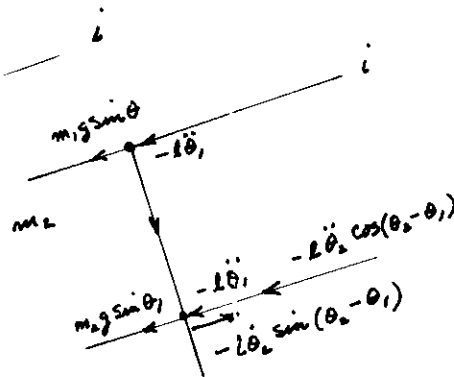
From $y_2=0$ $R = \frac{5}{18}P + \frac{1}{2}\frac{M}{l} + \frac{5}{6}\frac{M_2}{l}$ Subst. into above
 to obtain

$$\begin{Bmatrix} y_1 \\ \theta_1 \\ \theta_2 \end{Bmatrix} = \frac{1}{EI_1} \begin{bmatrix} 0.1018l^3 & 0.0833l^2 & -0.1944l^2 \\ 0.0833l^2 & 0.25l & -0.25l \\ -0.1944l^2 & -0.25l & 0.0916l \end{bmatrix} \begin{Bmatrix} P \\ M \\ M_2 \end{Bmatrix}$$

invert by computer program to obtain stiffness matrix (see Prob. 6-4)
 need numeric value of l .

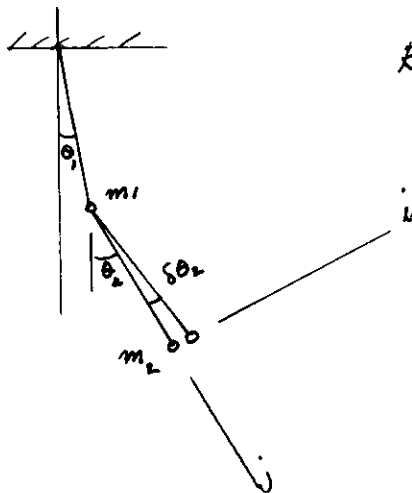


Resolve all forces including inertia forces
along i & j & dot product with $l\delta\theta_1 i$



$$\Sigma(F - m\ddot{u}) \cdot l\delta\theta_1 i = -(m_1 + m_2)g \sin \theta_1 - (m_1 + m_2)l\ddot{\theta}_1 - m_2 l \ddot{\theta}_2 \cos(\theta_2 - \theta_1) + m_2 l \dot{\theta}_2^2 \sin(\theta_2 - \theta_1) = 0$$

or $(m_1 + m_2)l\ddot{\theta}_1 + m_2 l \ddot{\theta}_2 \cos(\theta_2 - \theta_1) - m_2 l \dot{\theta}_2^2 \sin(\theta_2 - \theta_1) + (m_1 + m_2)g \sin \theta_1 = 0$



Resolve all forces along new i, j
 \perp & \parallel to l_2 and dot with $l\delta\theta_2 i$

$$m_2 l \ddot{\theta}_2 + m_2 l \ddot{\theta}_1 \cos(\theta_2 - \theta_1) + m_2 l \dot{\theta}_1^2 \sin(\theta_2 - \theta_1) + m_2 g \sin \theta_2 = 0$$

Try checking these equations from Lagrange's eqs.

$$\frac{d}{dt} \left(\frac{\partial T}{\partial \dot{\theta}_i} \right) - \frac{\partial T}{\partial \theta_i} + \frac{\partial U}{\partial \theta_i} = 0$$

$$\underline{8-1} \quad c = \sqrt{\frac{T}{\rho}} = \sqrt{\frac{444}{.372}} = 34.55 \text{ m/s}$$

$$\underline{8-2} \quad \text{see Sec. 8.1}$$

$$f_n = \frac{n}{2l} \sqrt{\frac{T}{\rho}}, \quad n = 1, 2, 3 \dots$$

8-3

$$\text{Gen sol. } y(x, t) = (A \sin \frac{\omega x}{c} + B \cos \frac{\omega x}{c}) \sin \omega t$$

$$\text{At } x=0 \quad y=0 \quad \therefore B=0$$

$$\text{At } x=l \quad y = y(l, t) \text{ of spring mass}$$

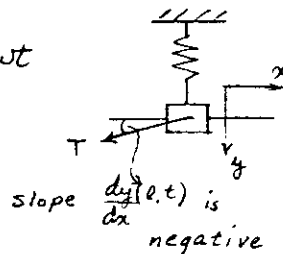
$$\text{Vertical force} = -T \frac{dy(l, t)}{dx}$$

$$= (-TA \frac{\omega}{c} \cos \frac{\omega l}{c}) \sin \omega t$$

$$= m \ddot{y}$$

$$\therefore y(l) = \frac{-TA \frac{\omega}{c} \cos \frac{\omega l}{c}}{k - m\omega^2} = \frac{-TA \frac{\omega}{c} \cos \frac{\omega l}{c}}{m\omega_m^2 (1 - \frac{\omega^2}{\omega_m^2})}, \quad \omega_m = \sqrt{\frac{k}{m}}$$

$$\text{or } \tan \frac{\omega l}{c} = -\left(\frac{T}{kl}\right) \left[\frac{(\frac{\omega l}{c})}{1 - (\frac{\omega l}{c})^2 (\frac{mc^2}{kl^2})} \right]$$



8-4

$$y_1 = a \cos kx \sin \omega t, \quad y_2 = a \cos\left(\frac{k}{2} + \frac{\pi}{2}\right) \sin\left(\omega t + \frac{T}{2}\right)$$

$$= -a \sin kx \cos \omega t$$

$$y = y_1 + y_2 = a [\cos kx \sin \omega t - \sin kx \cos \omega t]$$

$$= a \sin(\omega t - kx) = a \sin k \left(\frac{\omega}{k} t - x \right) \therefore c = \frac{\omega}{k}$$

8-5

$$c = \sqrt{\frac{E}{\rho}} = \sqrt{\frac{200 \times 10^9}{7810}} = 5060 \text{ m/s} = 16,600 \text{ ft/s}$$

8-6

$$dT \approx \rho g dx$$

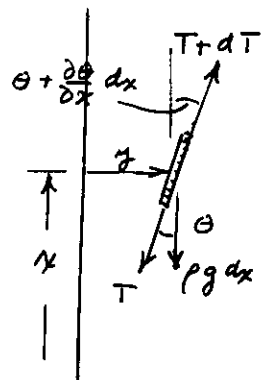
$$(T + dT)(\theta + \frac{\partial \theta}{\partial x} dx) - T\theta = \rho dx \ddot{y}$$

$$\therefore T \frac{\partial \theta}{\partial x} + \rho g \theta = \rho \ddot{y}$$

$$\theta = \frac{\partial y}{\partial x}, \quad T = \rho g x$$

$$\rho g x \frac{\partial^2 y}{\partial x^2} + \rho g \frac{\partial y}{\partial x} = \rho \ddot{y}$$

$$\ddot{y} = g \left(x \frac{\partial^2 y}{\partial x^2} + \frac{\partial y}{\partial x} \right)$$



8-7

$$y = Y(x) \cos \omega t$$

$$\ddot{y} = -\omega^2 Y(x) \cos \omega t$$

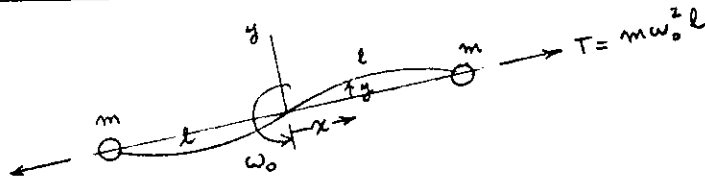
$$-\omega^2 Y(x) = g \left(x \frac{d^2 Y(x)}{dx^2} + \frac{dY(x)}{dx} \right)$$

$$\text{Let } z^2 = \frac{4\omega^2}{g} x \quad dx = \frac{g}{4\omega^2} 2z dz, \quad (dx)^2 = \left(\frac{g 2z}{4\omega^2} \right)^2 dz^2$$

$$-\omega^2 Y = g \left\{ \frac{g}{4\omega^2} z^2 \frac{d^2 Y}{\frac{g 2z}{4\omega^2} dz^2} + \frac{2\omega^2}{g z} \frac{dY}{dz} \right\}$$

$$\therefore \frac{d^2 Y}{dz^2} + \frac{1}{z} \frac{dY}{dz} + Y = 0 \quad \text{Bessel's D.E.}$$

8-8



Assume mode shape as shown. Accel. at y is $\ddot{y} - y\omega_0^2$ in lateral direction

$$T \frac{d^2 y}{dx^2} = \rho (\ddot{y} - y\omega_0^2) \quad \text{Let } y = Y(x) e^{i\omega t}$$

$$\frac{d^2 Y}{dx^2} + \left[\left(\frac{\omega}{c} \right)^2 + \left(\frac{\omega_0}{c} \right)^2 \right] Y(x) = 0 \quad c = \sqrt{\frac{T}{\rho}}$$

$$Y(0) = 0 \quad \therefore B = 0 \quad \text{and } Y(x) = A \sin \Omega x + 0$$

$$\text{where } \Omega = \sqrt{\left(\frac{\omega}{c} \right)^2 + \left(\frac{\omega_0}{c} \right)^2}$$

$$Y(l) = 0 \quad \therefore \sin \Omega l = 0$$

$$\sqrt{\left(\frac{\omega}{c} \right)^2 + \left(\frac{\omega_0}{c} \right)^2} l = \pi \quad \omega^2 = \left(\frac{\pi c}{l} \right)^2 - \omega_0^2$$

$$\therefore \underline{\underline{\omega^2 = \left(\frac{\pi}{l} \right)^2 \left(\frac{m\omega_0^2 l}{\rho} \right) - \omega_0^2}}$$

8-9

$$u = \left(A \sin \frac{\omega x}{c} + B \cos \frac{\omega x}{c} \right) \sin \omega t$$

$$u(0) = 0 \quad \therefore B = 0$$

$$\sigma = E \left(\frac{du}{dx} \right)_{x=l} = 0 \quad \therefore \frac{\omega}{c} \cos \frac{\omega l}{c} = 0$$

$$\frac{\omega l}{c} = \frac{\pi}{2}, \frac{3\pi}{2}, \frac{5\pi}{2} \dots = \left(n + \frac{1}{2} \right) \pi \quad n = 0, 1, 2, \dots$$

$$f_n = \frac{\omega_n}{2\pi} = \left(n + \frac{1}{2} \right) \frac{c}{2l}, \quad n = 0, 1, 2, \dots$$

8-10

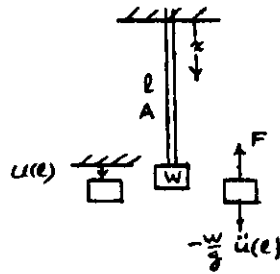
$$u = C \sin \frac{\omega x}{c} \sin \omega t$$

$$F = A\sigma = AE \frac{\partial u}{\partial x} = AEC \frac{\omega}{c} \cos \frac{\omega x}{c} \sin \omega t$$

$$= -\frac{W}{g} \ddot{u}(l)$$

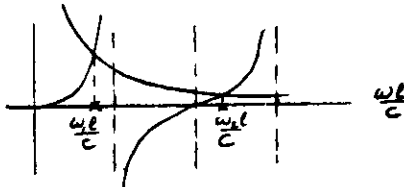
$$\therefore AE \frac{\omega}{c} \cos \frac{\omega l}{c} = \omega^2 \frac{W}{g} \sin \frac{\omega l}{c}$$

$$\frac{\omega l}{c} \tan \frac{\omega l}{c} = \frac{Apl}{W} = \frac{W_{rod}}{W_{end}}$$



8-11

From Prob. 8-10 $\tan \frac{\omega l}{c} = \frac{(\frac{m_{rod}}{M})}{\frac{\omega l}{c}}$



Let freq. eq. be $\omega_1 = n_1 \sqrt{\frac{Eg}{\rho_w}}$
where n_1 depends on end cond.

ρ_w = weight density
 ρ = mass "

$$\omega_1 = n_1 \sqrt{\frac{EA}{l} \frac{gl^2}{\rho Al}} = n_1 l \sqrt{\frac{EA}{l} \frac{g}{\rho Al}} = n_1 l \sqrt{\frac{k}{m_{rod}}}$$

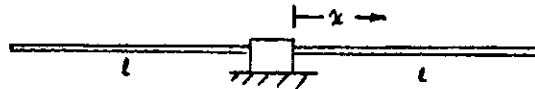
Let $n = \frac{m_{rod}}{M}$, then $\omega_1 = n_1 l \sqrt{\frac{k}{nM}}$

Approx sol.

$$\omega_{approx.} \approx \sqrt{\frac{AE/l}{M + \frac{1}{3}m_{rod}}} = \sqrt{\frac{k}{M + \frac{1}{3}M}}$$

$$\frac{\omega_{approx.}}{\omega_1} = \sqrt{\frac{k}{M(1 + \frac{1}{3}n)}} \sqrt{\frac{nM}{k} \left(\frac{1}{n_1 l} \right)} = \frac{1}{\beta_1} \sqrt{\frac{3n}{3+n}}$$

8-12



$$u(0) = 0 \quad \therefore u(x) = A \sin \omega \frac{x}{c}$$

$$\left(\frac{\partial u}{\partial x} \right)_{x=l} = 0 \quad \therefore \cos \frac{\omega l}{c} = 0 \quad \frac{\omega l}{c} = \frac{\pi}{2}, \frac{3\pi}{2}, \frac{5\pi}{2} \dots$$

$$\omega_1 = \frac{\pi}{l} \frac{c}{2} = \frac{\pi}{2l} \sqrt{\frac{Eg}{\rho_w}} = 2\pi (20,000)$$

$$l = \frac{1}{4 \times 20,000} \sqrt{\frac{Eg}{\rho_w}} = \frac{10^3}{80,000} \sqrt{\frac{30 \times 386}{0.31}} = 7.64'$$

Note $\rho = \frac{\rho_w}{g}$ = mass density

8-13

$$m \frac{\partial^2 u}{\partial t^2} = AE \frac{\partial^2 u}{\partial x^2} - \alpha \frac{\partial u}{\partial t} + \frac{P_0}{L} p(x) f(t)$$

$$u = \sum_i \phi_i(x) q_i(t) \quad \text{subst into eq. above}$$

$$m \sum_i \phi_i \ddot{q}_i = AE \sum_i \phi_i'' q_i - \alpha \sum_i \phi_i \dot{q}_i + \frac{P_0}{L} p(x) f(t)$$

mult. by $\phi_j dx$ and integrate over $x=0$ to l

$$m \int_0^l \phi_j \sum_i \phi_i \ddot{q}_i dx = AE \int_0^l \phi_j \sum_i \phi_i'' q_i dx - \alpha \int_0^l \phi_j \sum_i \phi_i \dot{q}_i dx + \frac{P_0}{L} \int_0^l p(x) \phi_j dx f(t)$$

Since ϕ_j and ϕ_i are orthogonal

$$\ddot{q}_j \int_0^l m \phi_j^2 dx = AE q_j \int_0^l \phi_j \phi_j'' dx - \alpha \dot{q}_j \int_0^l \phi_j^2 dx + \frac{P_0}{L} f(t) \int_0^l p(x) \phi_j dx$$

$$\ddot{q}_j + 25 \omega_j \dot{q}_j + \omega_j^2 q_j = \frac{P_0}{mL} f(t) \int_0^l p(x) \phi_j dx$$

$$\text{with } b_j = \frac{1}{L} \int_0^l p(x) \phi_j dx$$

$$\ddot{q}_j + 25 \omega_j \dot{q}_j + \omega_j^2 q_j = \frac{P_0}{m} b_j f(t)$$

From Eq. 8.2-1

$$q_j = \frac{P_0}{m} b_j \int_0^t f(t-\tau) e^{-5\omega_j \tau} \sin \omega_j \sqrt{1-5^2} \tau d\tau$$

$$u = \sum_j \phi_j q_j$$

8-14

From Eq. 8.3-3

$$c = \sqrt{\frac{2G}{\rho_w}}$$

for steel $G = 12 \times 10^6$ psi

$$\rho_w = 0.282 \text{ lb/in}^3$$

$$c = 10^3 \sqrt{\frac{12 \times 386}{0.282}} = 128,000 \text{ in/sec} = 10,660 \text{ ft/sec}$$

8-15



$$\theta = A \sin \frac{\omega x}{c}$$

$$\left(\frac{\partial \theta}{\partial x} \right)_{x=l/2} = 0 \quad \therefore \cos \frac{\omega l}{2c} = 0$$

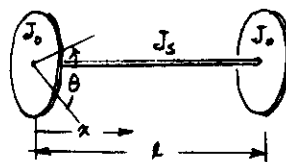
$$\frac{\omega l}{2c} = \frac{\pi}{2}, \frac{3\pi}{2}, \frac{5\pi}{2} \dots$$

$$\omega_m = \frac{\pi c}{l}, \frac{3\pi c}{l}, \dots = (2m-1) \frac{c}{l} \quad m=1, 2, 3 \dots$$

Ex-16

$$J_0 \ddot{\theta}_{x=0} = G I_p \left(\frac{d\theta}{dx} \right)_{x=0} = 0$$

$$J_0 \ddot{\theta}_{x=l} = -G I_p \left(\frac{d\theta}{dx} \right)_{x=l}$$



$$\theta = (A \sin \frac{\omega x}{c} + B \cos \frac{\omega x}{c}) \sin \omega t$$

$$-\omega^2 J_0 B = G I_p \frac{\omega}{c} A$$

$$-\omega^2 J_0 [A \sin \frac{\omega l}{c} + B \cos \frac{\omega l}{c}] = -G I_p \frac{\omega}{c} [A \cos \frac{\omega l}{c} - B \sin \frac{\omega l}{c}]$$

$$\therefore B = -\frac{G I_p}{\omega c J_0} A$$

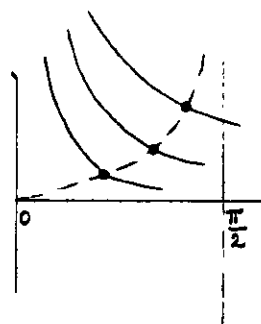
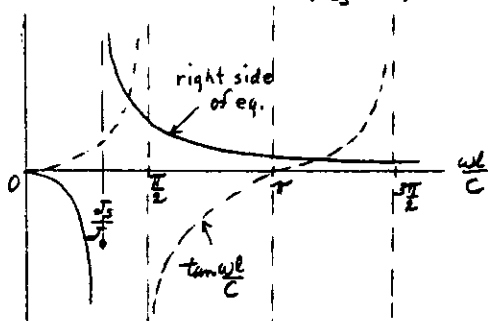
$$\sin \frac{\omega l}{c} - \frac{G I_p}{\omega c J_0} \cos \frac{\omega l}{c} = \frac{G I_p}{\omega c J_0} (\cos \frac{\omega l}{c} + \frac{G I_p}{\omega c J_0} \sin \frac{\omega l}{c})$$

$$\frac{G I_p}{\omega c J_0} = \frac{G I_p}{\omega c J_0} \frac{\rho l}{\rho l} \frac{g}{\rho l} = \frac{G g}{\rho} \frac{J_s}{J_0} \frac{1}{\omega c} = \frac{J_s}{J_0} \frac{c}{\omega l}$$

$$\therefore \left\{ \tan \frac{\omega l}{c} \right\} \left\{ 1 + \left(\frac{J_s}{J_0} \frac{c}{\omega l} \right)^2 \right\} = 2 \frac{J_s}{J_0} \frac{c}{\omega l}$$

$$\tan \frac{\omega l}{c} = \frac{2 \left(\frac{J_s}{J_0} \frac{\omega l}{c} \right)}{\left(\frac{J_s}{J_0} \frac{\omega l}{c} \right)^2 - 1}$$

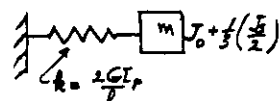
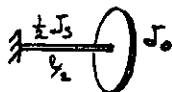
$$\frac{J_s}{J_0} < 1$$



When ends are free ($J_0 = 0$), $\frac{\omega l}{c} = \frac{\pi}{2}$. The effect of J_0 is to Lower nat. freq. If J_0/J_s is very large the right side of eq

$$\approx \frac{2}{\left(\frac{J_s}{J_0} \frac{\omega l}{c} \right)}$$

Fundamental freq. has node at center



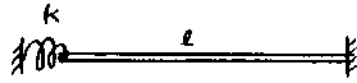
$$\omega_1 \approx \sqrt{\frac{2GI_p/l}{J_0 + \frac{1}{2}J_s}} = \sqrt{\frac{2Gg}{\rho} \frac{J_s/l^2}{J_0 + \frac{1}{2}J_s}} = \frac{c}{l} \sqrt{\frac{2J_s}{J_0 + \frac{1}{2}J_s}} = \frac{c}{l} \sqrt{\frac{2}{\frac{J_0}{J_s} + \frac{1}{2}}}$$

$$\text{When } \frac{J_0}{J_s} = 0 \quad \frac{\omega_1 l}{c} = \sqrt{3}$$

$$\text{When } \frac{J_0}{J_s} = 5, \text{ exact eq } \tan \frac{\omega l}{c} = \frac{10 \frac{\omega l}{c}}{25 \left(\frac{\omega l}{c} \right)^2 - 1} \text{ has root } \frac{\omega_1 l}{c} = 0.622$$

$$\text{Approx eq. } \frac{\omega_1 l}{c} = \sqrt{\frac{2}{5 + \frac{1}{2}}} \text{ gives } 0.62$$

8-17



$$\theta = (A \sin \frac{\omega x}{c} + B \cos \frac{\omega x}{c}) \sin \omega t$$

$$\theta(l) = 0 \quad \therefore A \sin \frac{\omega l}{c} + B \cos \frac{\omega l}{c} = 0, \quad B = -A \tan \frac{\omega l}{c}$$

$$\theta = A \left(\sin \frac{\omega x}{c} - \tan \frac{\omega l}{c} \cos \frac{\omega x}{c} \right) \sin \omega t$$

$$\text{Torque at } x=0 = K \theta(0) = K A (-\tan \frac{\omega l}{c})$$

$$\left(\frac{d\theta}{dx} \right)_{x=0} = A \frac{\omega}{c} (1)$$

$$G I_p \left(\frac{d\theta}{dx} \right)_{x=0} = A \frac{\omega}{c} G I_p = -K A \tan \frac{\omega l}{c}$$

$$\therefore \tan \frac{\omega l}{c} = - \frac{\omega I_p G}{K c} = - \frac{I_p G}{K l} \left(\frac{\omega l}{c} \right)$$

8-18

$$y = A \cosh \beta x + B \sinh \beta x + C \cos \beta x + D \sin \beta x$$

At $x=0$ and l , $\frac{d^2 y}{dx^2} + \frac{d^2 y}{dx^2} = 0$ so the 4 eqs become

$$A + 0 - C + 0 = 0$$

$$0 + B + 0 - D = 0$$

$$A \cosh \beta l + B \sinh \beta l - C \cos \beta l - D \sin \beta l = 0$$

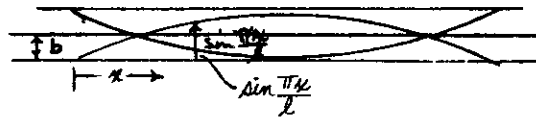
$$A \sinh \beta l + B \cosh \beta l + C \sin \beta l - D \cos \beta l = 0$$

$$\therefore C = A \quad \text{and} \quad D = B$$

$$-\frac{A}{B} = \frac{(\sinh \beta l - \sin \beta l)}{(\cosh \beta l - \cos \beta l)} = \frac{(\cosh \beta l - \cos \beta l)}{(\sinh \beta l + \sin \beta l)}$$

$$\therefore \text{freq. eq. becomes } \cosh \beta l \cos \beta l = 1.0$$

8-19



Momentum =

$$m \int_0^l (\sin \frac{\pi x}{l} - b) dx = 0 \quad \text{gives} \quad b = \frac{2}{\pi}$$

$$T = \frac{1}{2} m \omega^2 \int_0^l (\sin \frac{\pi x}{l} - b)^2 dx = \frac{1}{2} m \omega^2 \left[\frac{l}{2} - \frac{8}{\pi^2} l + \frac{4}{\pi^2} l \right] = \frac{1}{2} m \omega^2 l \left(\frac{1}{2} - \frac{4}{\pi^2} \right)$$

$$U = \frac{1}{2} EI \int_0^l \left(\frac{dy}{dx} \right)^2 dx = \frac{1}{2} EI \left(\frac{\pi}{l} \right)^3 \int_0^l \frac{1}{2} (1 - \cos 2 \frac{\pi x}{l}) dx = \frac{1}{2} EI \left(\frac{\pi}{l} \right)^3 \frac{l}{2}$$

$$\text{Equate } T \text{ \& } U \quad \omega_1^2 = \frac{\pi^4}{\pi^2 - 8} \left(\frac{EI}{ml^3} \right) = 5/2 \left(\frac{EI}{ml^3} \right)$$

$$\omega_1 = 24.6 \sqrt{\frac{EI}{ml^3}} \quad \text{node at } \left(\sin \frac{\pi x}{l} - \frac{2}{\pi} \right) = 0 \quad \text{or} \quad \frac{x}{l} = 0.22$$

8-20

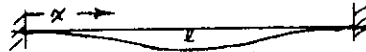
$$2\pi f_1 = 22.4 \sqrt{\frac{EI}{ml^3}} = 2\pi \cdot 1690$$

$$m = 2 \times 2 \times 1 \times \frac{153}{1732} \times \frac{1}{386} = 916 \times 10^{-6}$$

$$I = \frac{2 \times 2^3}{12} = \frac{4}{3} \quad \frac{EI}{ml^3} = \left(\frac{2\pi \times 1690}{22.4} \right)^2 = 224,000$$

$$E = \frac{224,000 \times 916 \times 10^{-6} \times 12^4}{\frac{4}{3}} = 3,480,000 \text{ lb/in}^2$$

8-21



Start with Eq B.4-12, At $x=0$ and $x=l$, $y = \frac{dy}{dx} = 0$

$$\left. \begin{array}{l} A + 0 + C + 0 = 0 \\ 0 + B + 0 + D = 0 \end{array} \right\} \quad \begin{array}{l} C = -A \\ D = -B \end{array}$$

$$A(\cosh \beta l - \cos \beta l) + B(\sinh \beta l - \sin \beta l) = 0$$

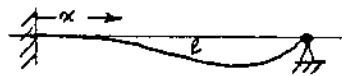
$$A(\sinh \beta l + \sin \beta l) + B(\cosh \beta l - \cos \beta l) = 0$$

$$-\frac{A}{B} = \frac{\sinh \beta l - \sin \beta l}{\cosh \beta l - \cos \beta l} = \frac{\cosh \beta l - \cos \beta l}{\sinh \beta l + \sin \beta l}$$

$$\text{or} \quad \cosh \beta l \cdot \cos \beta l = 1$$

8-22

Start with Eq. 8.4-12



At $x=0$, $y = \frac{dy}{dx} = 0$ gives $C = -A$
 $D = -B$

At $x=l$ $y = \frac{d^2y}{dx^2} = 0$

$$A(\cosh \beta l - \cos \beta l) + B(\sinh \beta l - \sin \beta l) = 0$$

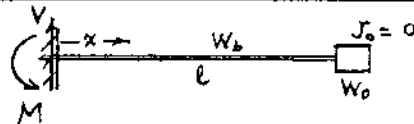
$$A(\cosh \beta l + \cos \beta l) + B(\sinh \beta l + \sin \beta l) = 0$$

$$-\frac{A}{B} = \frac{\sinh \beta l - \sin \beta l}{\cosh \beta l - \cos \beta l} = \frac{\sinh \beta l + \sin \beta l}{\cosh \beta l + \cos \beta l}$$

$$\therefore \cosh \beta l \sinh \beta l - \sinh \beta l \cos \beta l = 0$$

$$\text{or } \tanh \beta l = \tan \beta l$$

8-23



At $x=0$, $y = \frac{dy}{dx} = 0$

$\therefore C = -A, D = -B$

At $x=l$ $-EI \frac{d^3y}{dx^3} = -V = \frac{W_b}{g} \ddot{y}(l)$

$$-\beta^3 [A(\sinh \beta l - \sin \beta l) + B(\cosh \beta l + \cos \beta l)] = -\frac{W_b}{g} \frac{W_b}{EI} [A(\cosh \beta l - \cos \beta l) + B(\sinh \beta l - \sin \beta l)]$$

At $x=l$ $-M = -EI \frac{d^2y}{dx^2} = J_0 \left(\frac{dy}{dx} \right) = 0$

$$\beta^2 [A(\cosh \beta l + \cos \beta l) + B(\sinh \beta l + \sin \beta l)] = 0$$

$$-\frac{A}{B} = \frac{(\cosh \beta l + \cos \beta l) - \frac{W_b W_b}{\beta^3 g EI} (\sinh \beta l - \sin \beta l)}{(\sinh \beta l - \sin \beta l) - \frac{W_b W_b}{\beta^3 g EI} (\cosh \beta l - \cos \beta l)} = \frac{(\sinh \beta l + \sin \beta l)}{(\cosh \beta l + \cos \beta l)}$$

$$\beta^2 = \omega \sqrt{\frac{W_b l}{g EI l}} = \omega \sqrt{\frac{W_b}{g EI l}}$$

$$\therefore (1 - \cosh \beta l \cos \beta l) = \frac{W_b}{W_b} \beta l (-\cosh \beta l \sinh \beta l + \sinh \beta l \cos \beta l)$$

8-24

$$\text{At } x=0, y=y_0 \therefore y_0 = A+C$$

$$\text{At } x=0 \quad \frac{d^2 y}{dx^2} = 0 \therefore C=A$$

$$\text{At } x=l \quad \frac{d^2 y}{dx^2} = 0$$

$$B^3 [A(\cosh pl - \cos pl) + B \sinh pl - D \sin pl] = 0$$

$$\text{At } x=l \quad \frac{d^3 y}{dx^3} = 0$$

$$B^3 [A(\sinh pl + \sin pl) + B \cosh pl - D \cos pl] = 0$$

$$\text{At } x=l, y=y_l$$

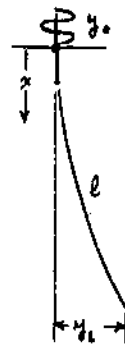
$$y_l = A(\cosh pl + \cos pl) + B \sinh pl + D \sin pl$$

$$\frac{y_0}{y_l} = \frac{2A}{A(\cosh l + \cos l) + B \sinh l + D \sin l} = \frac{2A}{A(\cosh l + \cos l) + B \sinh l + A(\cosh l - \cos l) + B \sinh l}$$

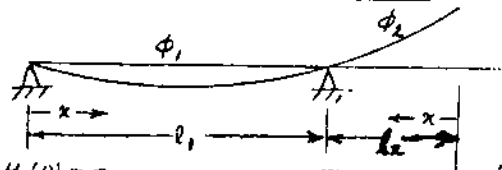
$$\text{where } Ch = \cosh pl, C = \cos pl, Sh = \sinh pl, S = \sin pl$$

$$\text{Also } y_0 = 2A$$

$$\frac{y_0}{y_l} = \frac{\sinh pl \cos pl - \cosh pl \sin pl}{\sinh pl - \sin pl}$$



8-25



$$\begin{array}{ll} \text{Boundary} & y_1(0) = 0 \\ \text{intermediate cond.} & \frac{d^2 y_1(0)}{dx^2} = 0 \end{array} \quad \begin{array}{ll} y_1(l_1) = 0 & y_1'(l_1) = y_2'(l_2) \\ y_2''(0) = 0 & y_2'''(0) = 0 \end{array}$$

$$\left. \begin{array}{l} y_1(0) = A_1 + C_1 = 0 \quad \therefore C_1 = -A_1 \\ y_1''(0) = A_1 - C_1 = 0 \quad \therefore C_1 = +A_1 \end{array} \right\} \therefore A_1 = C_1 = 0$$

$$y_1 = B_1 \sinh px + D_1 \sin px$$

$$y_1(l_1) = B_1 \sinh pl_1 + D_1 \sin pl_1 = 0 \quad \therefore B_1 = -D_1 \frac{\sin pl_1}{\sinh pl_1}$$

$$y_1 = D_1 \left(\sin px - \frac{\sin pl_1}{\sinh pl_1} \sinh px \right) = \phi_1(x)$$

$$y_2''(0) = A_2 - C_2 = 0 \quad \therefore C_2 = A_2$$

$$y_2'''(0) = B_2 - D_2 = 0 \quad \therefore D_2 = B_2$$

$$y_2(l_2) = A_2(\cosh pl_2 + \cos pl_2) + B_2(\sinh pl_2 + \sin pl_2)$$

$$\therefore B_2 = -A_2 \frac{\cosh pl_2 + \cos pl_2}{\sinh pl_2 + \sin pl_2}$$

$$y_2 = A_2 \left\{ (\cosh px + \cos px) - \left(\frac{\cosh pl_2 + \cos pl_2}{\sinh pl_2 + \sin pl_2} \right) (\sinh px + \sin px) \right\} = \phi_2(x)$$

$$\underline{9-1} \quad m \ddot{x} + kx = P_0$$

$$x = \frac{P_0}{k} (1 - \cos \omega t) \quad , \quad \omega = \sqrt{\frac{k}{m}}$$

$$\therefore x_{\max} = 2 \frac{P_0}{k}$$

9-2

$$\ddot{q}_i + 2\zeta_i \omega_i \dot{q}_i + \omega_i^2 q_i = \frac{P_0}{m}$$

$$q_i = \frac{P_0}{m} \left[1 - \frac{e^{-\zeta_i \omega_i t}}{\sqrt{1-\zeta_i^2}} \cos(\sqrt{1-\zeta_i^2} \omega_i t - \gamma_i) \right]$$

$$\tan \gamma_i = \frac{\zeta_i}{\sqrt{1-\zeta_i^2}} \quad \text{For small damping } \gamma_i \approx 0$$

$$\sqrt{1-\zeta_i^2} \approx 1$$

$$\therefore q_i \approx \frac{P_0}{k} [1 - e^{-\zeta_i \omega_i t} \cos \omega_i t] = \frac{P_0}{k} D_i(t)$$

9-3

$$\ddot{q}_i + 2\zeta_i \omega_i \dot{q}_i + \omega_i^2 q_i = \frac{f(t)}{M} \int_0^l \frac{P_0}{l} \phi_i(x) dx$$

$$\text{where } f(x, t) = \frac{P_0}{l} f(t) \quad \text{and } P_0 = \text{total force}$$

$$\text{Mode participation factor} = \frac{1}{l} \int_0^l \phi_i(x) dx$$

9-4

$$\text{Generalized force} = \frac{P_0}{M_i} \left\{ \frac{1}{l} \int_0^l p(x) \phi_i(x) dx \right\} f(t)$$

(See Sec. 9.1)

$$K_i = \frac{1}{l} \int_0^l p(x) \phi_i(x) dx \quad , \quad p(x) = l \delta(x-a)$$

$$\therefore K_i = \frac{1}{l} \int_0^l l \delta(x-a) \phi_i(x) dx = \phi_i(a)$$

$$q_i = \frac{P_0 K_i}{M_i \omega_i^2} D_i(t) \quad , \quad y(x, t) = \sum_i q_i \phi_i = \sum_i \frac{P_0 \phi_i(a) \phi_i(x)}{M_i \omega_i^2} D_i(t)$$

$$\omega_i^2 = (\beta_i l)^2 \frac{EI}{M_i l^3} \quad \therefore y(x, t) = \frac{P_0 l^3}{EI} \sum_i \frac{\phi_i(a) \phi_i(x)}{(\beta_i l)^4} D_i(t)$$

$$\begin{aligned}
 \underline{9-5} \quad K_i &= \lim_{\epsilon \rightarrow 0} \frac{1}{l} \int_0^l l^2 \left[\frac{\delta(x-a-\epsilon) - \delta(x-a)}{\epsilon} \right] \phi_i(x) dx \\
 &= \lim_{\epsilon \rightarrow 0} l \left[\frac{\phi_i(a+\epsilon) - \phi_i(a)}{\epsilon} \right] = l \left. \frac{d\phi_i(x)}{dx} \right|_{x=a} \\
 &= \beta_i l \cdot \frac{1}{\beta_i} \left. \frac{d\phi_i(x)}{dx} \right|_{x=a} = \beta_i l \phi_i'(a) \\
 &\quad \text{See Append. D for } \phi'
 \end{aligned}$$

$$\begin{aligned}
 \underline{9-6} \quad K_i &= \frac{1}{l} \int_0^l l \delta(x - \frac{l}{2}) \phi_i(x) dx = \phi_i(\frac{l}{2}) \\
 \phi_n(x) &= \sqrt{2} \sin n\pi \frac{x}{l} \quad \text{when normalized to total mass } M \\
 \phi_n(\frac{l}{2}) &= \sqrt{2} \sin \frac{n\pi}{2}, \quad n = 1, 3, 5, \dots \\
 \omega_n^2 &= (\beta_n l)^4 \frac{EI}{M l^3}, \quad \beta_n = \frac{n\pi}{l}, \quad n = 1, 3, 5, \dots \\
 y(x, t) &= \frac{2 P_0 l^3}{EI} \sum_n \left\{ \frac{\sin \frac{n\pi}{2} \cdot \sin n\pi \frac{x}{l}}{(n\pi)^4} D_n(t) \right\} \quad n = 1, 3, 5, \dots
 \end{aligned}$$

$$\begin{aligned}
 \underline{9-7} \quad &\text{From Prob. 9-5} \quad K_n = \beta_n l \phi_n'(a), \quad \beta_n = \frac{n\pi}{l} \\
 \phi_n(x) &= \sqrt{2} \sin n\pi \frac{x}{l} \quad \frac{d\phi_n}{dx} = \sqrt{2} \frac{n\pi}{l} \cos n\pi \frac{x}{l} \\
 \phi_n'(\frac{l}{2}) &= \frac{1}{\beta_n} \frac{d\phi_n(\frac{l}{2})}{dx} = \frac{\sqrt{2}}{l} \cos \frac{n\pi}{2}, \quad n = 2, 4, 6, \dots \\
 \ddot{q}_n + \omega_n^2 q_n &= \frac{m_0}{M} [\beta_n l \phi_n'(a)] f(t) \\
 q_n &= \frac{m_0}{M \omega_n^2} \beta_n l \phi_n'(\frac{l}{2}) D_n(t) \\
 y(x, t) &= \frac{m_0}{M} \sum_n \frac{\beta_n l}{\omega_n^2} \phi_n(\frac{l}{2}) \phi_n(x) D_n(t) = \frac{m_0 l^3}{EI} \sum_n \frac{\phi_n'(\frac{l}{2}) \phi_n(x)}{(\beta_n l)^3} D_n(t) \\
 &= \frac{2 m_0 l^2}{EI} \sum_{n=2,4,\dots} \frac{\cos \frac{n\pi}{2} \sin n\pi \frac{x}{l}}{(n\pi)^3} D_n(t)
 \end{aligned}$$

9-8 Only even modes, $n = 2, 6, \dots$ will give $K_n \neq 0$

$$\phi_n = \sqrt{2} \sin \frac{n\pi x}{l}$$

$$K_n = \frac{1}{l} \int_0^{l/2} \sqrt{2} \sin \frac{n\pi x}{l} dx - \frac{1}{l} \int_{l/2}^l \sqrt{2} \sin \frac{n\pi x}{l} dx = \frac{2\sqrt{2}}{n\pi} (1 - \cos \frac{n\pi}{2})$$

$$D_n = \omega_n \int_0^t \sin \omega_n (t-\xi) d\xi = 1 - \cos \omega_n t$$

$$y(x, t) = \sum_n \frac{p_0 l K_n \phi_n}{\omega_n^2 M} D_n = \sum_{n=2, 6, \dots} \frac{p_0 l}{\omega_n^2 M} \frac{2\sqrt{2}}{n\pi} (1 - \cos \frac{n\pi}{2}) \sqrt{2} \sin \frac{n\pi x}{l} (1 - \cos \omega_n t)$$

$$= \frac{2p_0 l}{\pi M} \sum_{n=2, 6, \dots} 2 (1 - \cos \frac{n\pi}{2}) \sin \frac{n\pi x}{l} (1 - \cos \omega_n t)$$

1st mode, $n = 2$

$$y_1(x, t) = \frac{4p_0 l}{\pi M \omega_2^2} \sin \frac{2\pi x}{l} (1 - \cos \omega_2 t)$$

2nd mode, $n = 6$

$$y_2(x, t) = \frac{4p_0 l}{\pi M \omega_6^2} \sin \frac{6\pi x}{l} (1 - \cos \omega_6 t)$$

9-9

Normal mode $u(x) = A \sin \frac{\omega x}{c} + B \cos \frac{\omega x}{c}$

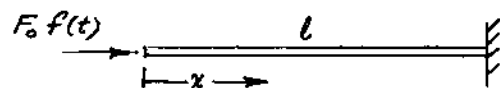
$$\frac{du}{dx} = \frac{\omega}{c} \left\{ A \cos \frac{\omega x}{c} - B \sin \frac{\omega x}{c} \right\}$$

$$u(x) = B \cos \frac{\omega x}{c}$$

$$\int_0^l \frac{M}{l} B^2 \cos^2 \frac{\omega x}{c} dx = M = \frac{B^2}{l} \int_0^l \frac{1}{2} (1 + \cos \frac{2\omega x}{c}) dx = \frac{B^2}{2}$$

$$\therefore B = \sqrt{2}$$

$$\phi_n(x) = \sqrt{2} \cos \frac{n\pi x}{2l}, \quad n = 1, 3, 5, \dots$$



Boundary Cond.

$$x = 0, \quad \frac{du}{dx} = 0 \quad \therefore A = 0$$

$$x = l, \quad u = 0 \quad \therefore B \cos \frac{\omega l}{c} = 0$$

$$\frac{\omega l}{c} = \frac{\pi}{2}, \quad \frac{3\pi}{2}, \quad \frac{5\pi}{2}, \quad \dots$$

9 - 9 Cont:

$$\ddot{q}_n + \omega_n^2 q_n = \frac{f(t)}{M} \int_0^l F_0 \delta(x) \sqrt{2} \cos \frac{n\pi x}{2l} dx = \frac{F_0 \sqrt{2}}{M} f(t)$$

$$q_n = \frac{F_0 K_n}{\omega_n^2 M} D_n(t) = \frac{\sqrt{2} F_0}{\left(\frac{n\pi}{2} \frac{c}{l}\right)^2 M} D_n(t) = \frac{\sqrt{2} F_0 M l^2}{\left(\frac{n\pi}{2}\right)^2 E A} D_n(t)$$

$$u(x, t) = \sum_n \phi_n(x) q_n(t) = \frac{2F_0 l}{EA} \left\{ \frac{\cos \frac{\pi x}{2l} \cdot D_1(t)}{\left(\frac{\pi}{2}\right)^2} + \frac{\cos \frac{3\pi x}{2l} \cdot D_3(t)}{\left(\frac{3\pi}{2}\right)^2} + \dots \right\}$$

9 - 10

$$K_n = \frac{1}{l} \int_0^l l \delta(x - \frac{l}{3}) \sqrt{2} \cos \frac{n\pi x}{2l} dx = \sqrt{2} \cos \frac{n\pi}{6}$$

$$n = 1, 3, 5, \dots$$

n

$$1 \quad \cos \frac{\pi}{6} = \cos 30^\circ = .866$$

$$3 \quad \cos \frac{3\pi}{6} = \cos 90^\circ = 0 \quad \therefore \text{mode absent}$$

$$5 \quad \cos \frac{5\pi}{6} = \cos 150^\circ = -.866$$

$$7 \quad \cos \frac{7\pi}{6} = \cos 210^\circ = -.866$$

$$9 \quad \cos \frac{9\pi}{6} = \cos 270^\circ = 0 \quad \therefore \text{mode absent}$$

$$11 \quad \cos \frac{11\pi}{6} = \cos 330^\circ = .866$$

\therefore modes present are 1, 5, 7, 11, 13, ...

9 - 11

$$K_n = \sqrt{2} \cos \frac{n\pi}{6}$$

$$q_n = \frac{F_0 K_n}{\omega_n^2 M} D_n(t)$$

$$\omega_n^2 = \left(\frac{n\pi}{2} \frac{c}{l}\right)^2 = \left(\frac{n\pi}{2}\right)^2 \frac{EA}{Ml}$$

$$u(x, t) = \frac{2F_0 l}{AE} \left\{ \frac{0.866}{\left(\frac{\pi}{2}\right)^2} \cos \frac{\pi x}{2l} \cdot D_1(t) - \frac{0.866}{\left(\frac{5\pi}{2}\right)^2} \cos \frac{5\pi x}{2l} \cdot D_5(t) \right.$$

$$\left. - \frac{0.866}{\left(\frac{7\pi}{2}\right)^2} \cos \frac{7\pi x}{2l} \cdot D_7(t) + \frac{0.866}{\left(\frac{11\pi}{2}\right)^2} \cos \frac{11\pi x}{2l} \cdot D_{11}(t) + \dots \right\}$$

$$\frac{9-12}{2} \quad y(x,t) = q_1 \sin \frac{\pi x}{L} + q_2$$

$$T = \frac{1}{2} \int_0^L m \dot{y}^2 dx = \frac{1}{2} \int_0^L (\dot{q}_1 \sin \frac{\pi x}{L} + \dot{q}_2)^2 dx$$

$$= \frac{1}{2} m L \left[\frac{1}{L} \dot{q}_1^2 + \frac{4}{\pi^2} \dot{q}_1 \dot{q}_2 + \dot{q}_2^2 \right]$$

$$U = \frac{1}{2} \left(\frac{k}{L} \right) y^2(0) + \frac{1}{2} \left(\frac{k}{L} \right) y^2(L) + \frac{1}{2} \int_0^L EI y''^2 dx$$

$$= \frac{1}{2} k q_2^2 + \frac{1}{2} EI q_1^2 \int_0^L \left(\frac{\pi}{L} \right)^4 \sin^2 \frac{\pi x}{L} dx$$

$$= \frac{1}{2} k q_2^2 + \frac{1}{2} EI q_1^2 \left(\frac{\pi}{L} \right)^4 \frac{L}{2} \quad \text{Sub. into } \frac{d}{dt} \left(\frac{\partial T}{\partial \dot{q}_i} \right) + \frac{\partial U}{\partial q_i} = 0$$

$$\ddot{q}_1 + \frac{4}{\pi^2} \ddot{q}_2 + \pi^4 \frac{EI}{mL^2} q_1 = 0, \quad \text{and} \quad \frac{2}{\pi} \ddot{q}_1 + \ddot{q}_2 + \frac{k}{mL} q_2 = 0$$

rewrite

$$\ddot{q}_1 + \frac{4}{\pi^2} \ddot{q}_2 + \omega_{11}^2 q_1 = 0 \quad \text{and} \quad \frac{2}{\pi} \ddot{q}_1 + \ddot{q}_2 + \omega_{22}^2 q_2 = 0$$

$$\left. \begin{aligned} (\omega_{11}^2 - \omega^2) q_1 - \frac{4}{\pi} \omega^2 q_2 &= 0 \\ -\frac{2}{\pi} \omega^2 q_1 + (\omega_{22}^2 - \omega^2) q_2 &= 0 \end{aligned} \right\} \quad \omega^4 - \left(\frac{\pi^2}{\pi^2 - 8} \right) (\omega_{11}^2 + \omega_{22}^2) \omega^2 + \frac{\pi \omega_{11}^2 \omega_{22}^2}{\pi^2 - 8} = 0$$

$$\omega^2 = \frac{\omega_{22}^2}{2} \left(\frac{\pi^2}{\pi^2 - 8} \right) \left[(1+R) \pm \sqrt{(1-R)^2 + \frac{32}{\pi^2} R} \right] \quad \text{where } R = \left(\frac{\omega_{11}}{\omega_{22}} \right)^2$$

$$\text{Assume } y = (b + \sin \frac{\pi x}{L}) q$$

$$T = \frac{1}{2} m \dot{q}^2 \int_0^L (b^2 + 2b \sin \frac{\pi x}{L} + \sin^2 \frac{\pi x}{L}) dx = \frac{1}{2} m \dot{q}^2 \left(b^2 L + 4b \frac{L}{\pi} + \frac{L}{2} \right)$$

$$U = \left[\frac{1}{2} k b^2 + \frac{1}{2} EI \left(\frac{\pi}{L} \right)^4 \frac{L}{2} \right] q^2$$

$$T = U \quad \text{gives} \quad \omega^2 = \frac{\frac{k}{m} b^2 + \frac{1}{2} \omega_{11}^2}{b^2 + \frac{4b}{\pi} + \frac{1}{2}}$$

Momentum = 0 gives

$$kb - m\omega^2 \int_0^L (\sin \frac{\pi x}{L} + b) dx = 0 \quad \therefore kb - m\omega^2 \left(-2 \frac{L}{\pi} + bL \right) = 0$$

$$b = \frac{2}{\pi} \left(\frac{\omega^2}{\omega_{22}^2 - \omega^2} \right) = q_2/q_1, \quad \text{or} \quad \omega^2 = \frac{\omega_{22}^2 b}{b + \frac{2}{\pi}}$$

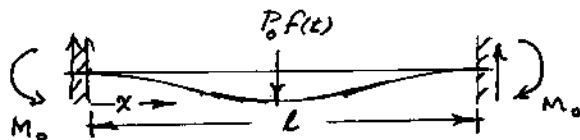
$$\text{Equating } \omega^2 \quad \frac{\omega_{22}^2 b}{b + \frac{2}{\pi}} = \frac{\omega_{22}^2 b^2 + \frac{1}{2} \omega_{11}^2}{b^2 + \frac{4b}{\pi} + \frac{1}{2}} \quad \therefore b^2 + \frac{\pi}{4} (1-R)b + \frac{1}{2} R = 0$$

$$b = \frac{\pi}{8} \left[(R-1) \pm \sqrt{(R-1)^2 + \frac{32}{\pi^2} R} \right]$$

$$R = \left(\frac{\omega_{11}}{\omega_{22}} \right)^2$$

9-13

$$y = \sum_n \phi_n q_n$$



Generalized force

$$= P_0 f(t) \int_0^l \phi_n(x) \delta(x - \frac{l}{2}) dx = P_0 \phi_n(\frac{l}{2}) f(t)$$

$$y(\frac{l}{2}, t) = \sum_n \phi_n(\frac{l}{2}) q_n(t) \quad \text{where } q_n = \text{solution of}$$

$$\ddot{q}_n + \omega_n^2 q_n = \frac{P_0}{M} \phi_n(\frac{l}{2}) f(t)$$

Solution

$$q_n(t) = q_n(0) \cos \omega_n t + \frac{1}{\omega_n} \dot{q}_n(0) \sin \omega_n t + \frac{P_0 \phi_n(\frac{l}{2})}{M \omega_n^2} \omega_n \int_0^t f(\xi) \sin \omega_n (t - \xi) d\xi$$

From Appendix D

$$\phi_1(\frac{l}{2}) = 1.583,$$

$$\beta_1 l = 4.73$$

$$\phi_1''(0) = 2.0$$

$$\phi_2(\frac{l}{2}) = 0$$

$$\beta_2 l = 7.85$$

$$\phi_2''(0) = 2.0$$

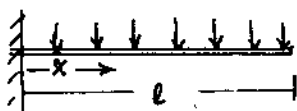
$$\phi_3(\frac{l}{2}) = -1.372$$

$$\beta_3 l = 10.99$$

$$\phi_3''(0) = 2.0$$

$$M_0 = EI \left(\frac{d^2 y}{dx^2} \right)_0 = EI \sum_n q_n(t) \left(\frac{d^2 \phi_n}{dx^2} \right)_{x=0} = EI \sum_n q_n(t) \beta_n^2 \phi_n''(0)$$

9-14



$$K_i = \frac{1}{l} \int_0^l \phi_i(x) dx$$

The above integral for K_i can be evaluated from tables in App. D. Add column of $\phi_i(x)$ and mult. by interval $\frac{dx}{l} = 0.04$. 1st mode column sum is 20.57 $\therefore 20.57 \times 0.04 = 0.822$. Exact value from integrating Eq. 8.4-12 with proper boundary conditions is 0.783. i.e. use $y(x) = (\cosh \beta x - \cos \beta x) - \left(\frac{\sinh \beta l - \sin \beta l}{\cosh \beta l + \cos \beta l} \right) (\sinh \beta x - \sin \beta x)$

Note: The number 20.57 is too large since it was computed with the largest deflection of $2.0 \times 0.04 + 1.89 \times 0.04 + \dots$ etc. If the sum is started with $1.89 \times 0.04 + \dots$ etc, then a figure of 18.57 is obtained. Averaging $\frac{1}{2}(18.57 + 20.57) \times 0.04$ we obtain the exact value 0.783.

Letting $\alpha_n = \frac{\sinh \beta l - \sin \beta l}{\cosh \beta l + \cos \beta l}$, the integral can be shown to

$$\text{be } \int_0^l \phi_n(x) \frac{dx}{l} = \frac{2\alpha_n}{\beta_n l}$$

9-15 From Eq. 9.3-8 $\cancel{\frac{1}{M}} M(\omega_1^2 - \omega^2) = -k \phi_1^2(\frac{l}{3}) \cancel{\frac{1}{M}}$

$$1 - (\frac{\omega}{\omega_1})^2 = -\frac{k}{M} \phi_1^2(\frac{l}{3}) \frac{1}{\omega_1^2} \quad (\frac{\omega}{\omega_1})^2 = 1 + \frac{k}{M} \phi_1^2(\frac{l}{3}) \frac{1}{\omega_1^2}$$

For simply supported beam $\phi_1(x) = \sqrt{2} \sin \frac{\pi x}{l}$

$$\therefore \phi_1^2(\frac{l}{3}) = 2 \sin^2 \frac{\pi}{3} = 1.50 \quad \omega_1^2 = \pi^4 \frac{EI}{Ml^3} \text{ for unconstrained beam.}$$

$$\therefore (\frac{\omega}{\omega_1})^2 = 1 + 1.5 \left(\frac{k}{M} \right) \left(\frac{Ml^3}{\pi^4 EI} \right)$$

9-16 From Eq. 9.3-8

$$\bar{q}_1 = \frac{1}{M(\omega_1^2 - \omega^2)} \left\{ -k \phi_1(\frac{l}{3}) \left[\bar{q}_1 \phi_1(\frac{l}{3}) + \bar{q}_2 \phi_2(\frac{l}{3}) \right] \right\}$$

$$\bar{q}_2 = \frac{1}{M(\omega_2^2 - \omega^2)} \left\{ -k \phi_2(\frac{l}{3}) \left[\bar{q}_1 \phi_1(\frac{l}{3}) + \bar{q}_2 \phi_2(\frac{l}{3}) \right] \right\}$$

Freq. Eq. becomes

$$\begin{vmatrix} [(\omega_1^2 - \omega^2) + \frac{k}{M} \phi_1^2(\frac{l}{3})] & \frac{k}{M} \phi_1(\frac{l}{3}) \phi_2(\frac{l}{3}) \\ \frac{k}{M} \phi_1(\frac{l}{3}) \phi_2(\frac{l}{3}) & [(\omega_2^2 - \omega^2) + \frac{k}{M} \phi_2^2(\frac{l}{3})] \end{vmatrix} = 0$$

9-17 We use here a somewhat different procedure than that of the text in that the influence coeffs. are found from the beam eq. $F(a, t) = -k y(\frac{l}{3})$

$$\alpha(a, x) = \alpha(\frac{l}{3}, \frac{l}{3}) = \frac{(\frac{l}{3})(\frac{2l}{3})}{6EI} (l^2 - \frac{4l^2}{9} - \frac{l^2}{9}) = \frac{4}{243} \frac{l^3}{EI}$$

From Eq. 9.4-4

$$y(\frac{l}{3}) = -k y(\frac{l}{3}) \frac{4}{243} \frac{l^3}{EI} + (\frac{\omega}{\omega_1})^2 q_1 \phi_1(\frac{l}{3}) + (\frac{\omega}{\omega_2})^2 q_2 \phi_2(\frac{l}{3}) + \dots$$

Using 1st mode only

$$y(\frac{l}{3}) \left[1 + \frac{4}{243} \frac{k l^3}{EI} \right] = (\frac{\omega}{\omega_1})^2 q_1 \phi_1(\frac{l}{3})$$

but $y(\frac{l}{3}) = \phi_1(\frac{l}{3}) q_1 + \text{higher modes which are neglected.}$

$$\therefore (\frac{\omega}{\omega_1})^2 = 1 + \frac{4}{243} \frac{k l^3}{EI}$$

Comparison with Prob. 9-15

$$\frac{1.5}{\pi^4} = 0.0154$$

$$\frac{4}{243} = 0.0165$$

9-18 From Eq. 9.4-4 with 1 mode

$$\phi_1(a) \bar{q}_1(t) = F(a, t) \alpha(a, a) - \left(\frac{\omega}{\omega_1}\right)^2 \bar{q}_1(t) \phi_1(a)$$

$$\phi_1(a) \bar{q}_1(t) = -k \phi_1(a) \bar{q}_1(t) \frac{\phi_1^2(a)}{M \omega_1^2} - \left(\frac{\omega}{\omega_1}\right)^2 \bar{q}_1(t) \phi_1(a)$$

$$\therefore \left(\frac{\omega}{\omega_1}\right)^2 = 1 + \frac{k}{M \omega_1^2} \phi_1^2(a)$$

9-19

$$\bar{q}_1(\omega_1^2 - \omega^2) = -\frac{k}{M} \varphi_1'(0) [\bar{q}_1 \varphi_1'(0) + \bar{q}_2 \varphi_2'(0)]$$

$$\bar{q}_2(\omega_2^2 - \omega^2) = -\frac{k}{M} \varphi_2'(0) [\bar{q}_1 \varphi_1'(0) + \bar{q}_2 \varphi_2'(0)]$$

Freq. eq. becomes

$$\begin{vmatrix} [(\omega_1^2 - \omega^2) + \frac{k}{M} \varphi_1'^2(0)] & \frac{k}{M} \varphi_1'(0) \varphi_2'(0) \\ \frac{k}{M} \varphi_1'(0) \varphi_2'(0) & [(\omega_2^2 - \omega^2) + \frac{k}{M} \varphi_2'^2(0)] \end{vmatrix} = 0$$

$\omega_2^2 = 16 \omega_1^2$ for simply supported beam. Let $\lambda = \left(\frac{\omega}{\omega_1}\right)^2$

$$[(1-\lambda) + \frac{k}{M \omega_1^2} \varphi_1'^2(0)] [(16-\lambda) + \frac{k}{M \omega_1^2} \varphi_2'^2(0)] - \left(\frac{k}{M \omega_1^2}\right)^2 [\varphi_1'(0) \varphi_2'(0)]^2 = 0$$

$$\lambda^2 - \left\{ 17 + \frac{k}{M \omega_1^2} [\varphi_1'^2(0) + \varphi_2'^2(0)] \right\} \lambda + \left\{ 16 + \frac{k}{M \omega_1^2} [\varphi_1'^2(0) + 16 \varphi_2'^2(0)] \right\} = 0$$

$$\varphi_1(x) = \sqrt{2} \sin \frac{\pi x}{\ell} \quad \varphi_1'(0) = \sqrt{2} \frac{\pi}{\ell} \quad \varphi_1'^2(0) = 2 \left(\frac{\pi}{\ell}\right)^2$$

$$\varphi_2(x) = \sqrt{2} \sin \frac{2\pi x}{\ell} \quad \varphi_2'(0) = 2\sqrt{2} \frac{\pi}{\ell} \quad \varphi_2'^2(0) = 8 \left(\frac{\pi}{\ell}\right)^2$$

If $k = 0$

$$\lambda^2 - 17\lambda + 16 = (\lambda - 1)(\lambda - 16) = 0$$

$$\therefore \begin{cases} \lambda_1 = 1 \\ \lambda_2 = 16 \end{cases}$$

If $k \neq 0$ Let $\alpha = \frac{k}{M \omega_1^2}$

$$\lambda^2 - [17 + 10 \left(\frac{\pi}{\ell}\right)^2 \alpha] \lambda + [16 + 40 \left(\frac{\pi}{\ell}\right)^2 \alpha] = 0$$

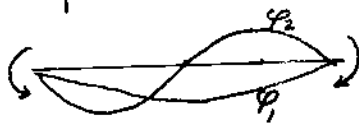
9-20

$$\bar{q}_1(\omega_1^2 - \omega^2) = -\frac{K}{M} \varphi_1'(0) [\bar{q}_1 \varphi_1'(0) + \bar{q}_2 \varphi_2'(0)] - \frac{K}{M} \varphi_1'(l) [\bar{q}_1 \varphi_1'(l) + \bar{q}_2 \varphi_2'(l)]$$

$$\bar{q}_2(\omega_2^2 - \omega^2) = -\frac{K}{M} \varphi_2'(0) [\bar{q}_1 \varphi_1'(0) + \bar{q}_2 \varphi_2'(0)] - \frac{K}{M} \varphi_2'(l) [\bar{q}_1 \varphi_1'(l) + \bar{q}_2 \varphi_2'(l)]$$

Free eq.

$$\begin{vmatrix} (\omega_1^2 - \omega^2) + \frac{K}{M} [\varphi_1'^2(0) + \varphi_1'^2(l)] & \frac{K}{M} [\varphi_1'(0) \varphi_2'(0) + \varphi_1'(l) \varphi_2'(l)] \\ \frac{K}{M} [\varphi_1'(0) \varphi_2'(0) + \varphi_1'(l) \varphi_2'(l)] & (\omega_2^2 - \omega^2) + \frac{K}{M} [\varphi_2'^2(0) + \varphi_2'^2(l)] \end{vmatrix} = 0$$



Since $\varphi_1'(0) = -\varphi_1'(l)$
 $\varphi_1'^2(0) = \varphi_1'^2(l)$
 $\varphi_2'^2(0) = \varphi_2'^2(l)$

$\varphi_1'(0) \varphi_2'(0) = 4\left(\frac{\pi}{l}\right)^2$ and $\varphi_1'(l) \varphi_2'(l) = -4\left(\frac{\pi}{l}\right)^2$ off diagonal terms are zero

$$[(1-\lambda) + \alpha 2 \varphi_1'^2(0)] [(16-\lambda) + \alpha 2 \varphi_2'^2(0)] = 0$$

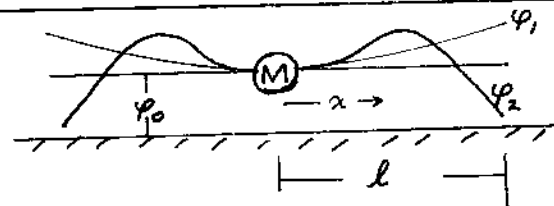
$$\lambda^2 - [17 + 2\alpha 10\left(\frac{\pi}{l}\right)^2] \lambda + [16 + 2\alpha 40\left(\frac{\pi}{l}\right)^2] = 0$$

$\therefore \alpha$ of Prob 9-19 is changed to 2α in Prob 9-20

9-21

$$\varphi_0 = 1$$

$\varphi_1(x)$ and $\varphi_2(x)$ etc are cantilever modes



$$y(x, t) = \varphi_0 q_0(t) + \sum_{n=1}^{\infty} \varphi_n(x) q_n(t) \quad \text{For free-free natural modes, momentum must be zero.}$$

$$\therefore 2 \int_0^l m y(x, t) dx + M_0 y(0, t) = 0$$

$$2 \sum_{n=1}^{\infty} q_n(t) \int_0^l m \varphi_n(x) dx + (M_0 + 2m) \varphi_0 q_0 = 0$$

If only 1 mode + translation is used $(M+2m)q_0 + 2q_1 m l(.783) = 0$

$$q_0 = -\left(\frac{2 \times .783 m l}{M_0 + 2m}\right) q_1$$

$$T = \frac{1}{2} \int_0^l 2m \dot{y}^2(x, t) dx + \frac{1}{2} M_0 \dot{y}^2(0, t)$$

$$= \int_0^l m [\varphi_0 \dot{q}_0 + \varphi_1 \dot{q}_1]^2 dx + \frac{1}{2} M_0 \varphi_0^2 \dot{q}_0^2$$

9-21 Cont:

$$T = \int_0^l m \dot{\varphi}_0^2 \dot{q}_0^2 dx + \int_0^l 2m \varphi_0 \dot{q}_0 \dot{q}_1 dx + \int_0^l m \varphi_1^2 \dot{q}_1^2 dx + \frac{1}{2} M_0 \dot{\varphi}_0^2 \dot{q}_0^2$$

$$= \frac{1}{2} \dot{q}_0^2 \{M_0 + 2ml\} + \{2ml \cdot 0.783\} \dot{q}_0 \dot{q}_1 + ml \dot{q}_1^2$$

$$U = \frac{1}{2} \int_0^l 2EI y''^2(x,t) dx = \frac{1}{2} \{2\omega_1^2 ml q_1^2\} \quad \text{see Eq. 9.1-7}$$

Lagrange's Eqs.

$$\ddot{q}_0 (M_0 + 2ml) + 2 \times 0.783 ml \ddot{q}_1 = 0$$

$$\ddot{q}_1 2ml + 2 \times 0.783 ml \ddot{q}_0 + 2\omega_1^2 ml q_1 = 0$$

Freq. Eq

$$\begin{vmatrix} -(M_0 + 2ml)\omega^2 & -2 \times 0.783 ml \omega^2 \\ -2 \times 0.783 ml \omega^2 & 2(\omega_1^2 - \omega^2) ml \end{vmatrix} = 0$$

Let $2ml = M$

$$-(M_0 + M)\omega^2(\omega_1^2 - \omega^2)M - (.783)^2 M^2 \omega^4 = 0$$

$$\frac{\omega}{\omega_1} = \sqrt{\frac{M_0 M + M^2}{M_0 M + 3.87 M^2}} \quad \text{where } \omega_1 = 1^{\text{st}} \text{ nat. freq. of cantilever beam of length } l \text{ and mass } \frac{1}{2} M$$

If $M_0 \rightarrow 0$, then $\omega = \omega_1 \frac{1}{\sqrt{3.87}} = 1.61 \omega_1$

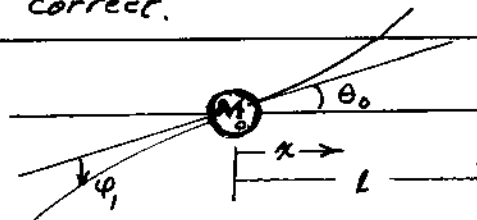
Since $\omega_1 = 3.52 \sqrt{\frac{EI}{ml^3}}$ and for free-free beam of length $2l$ has nat. freq $\omega_{ff} = 22.4 \sqrt{\frac{EI}{m(2l)^3}} = 5.57 \sqrt{\frac{EI}{ml^3}}$

$$\frac{5.57}{3.52} = 1.58. \quad \text{Since the airplane becomes a free-free beam of length } 2l \text{ as } M_0 \rightarrow 0, \text{ the above result is approximately correct.}$$

9-22

$$y(x,t) = q_0 \frac{x}{l} + \varphi_1 q_1$$

$$\theta_0 = \left(\frac{dy}{dx} \right)_{x=0} = \frac{q_0}{l}$$



$$T = \frac{1}{2} I_0 \left(\frac{\dot{q}_0}{l} \right)^2 + \frac{1}{2} \int_0^l 2m \left[\dot{q}_0 \frac{x}{l} + \varphi_1 \dot{q}_1 \right]^2 dx$$

$$= \frac{1}{2} I_0 \left(\frac{\dot{q}_0}{l} \right)^2 + \frac{1}{3} ml \dot{q}_0^2 + 2m \dot{q}_0 \dot{q}_1 \int_0^l \frac{x}{l} \varphi_1(x) dx + m \dot{q}_1^2 \int_0^l \varphi_1^2 dx$$

9-22 Cont.

$$U = \frac{1}{2} \{ 2\omega_1^2 (ml) \dot{q}_1^2 \} \quad \text{see Eq. 9.1-7}$$

Lagrange's Eq. & characteristic eq.

$$\begin{vmatrix} -\left(\frac{I_0}{l^2} + \frac{2ml}{3}\right)\omega^2 & -\frac{2m\omega^2}{l} \int_0^l x \varphi_1 dx \\ -\frac{2m\omega^2}{l} \int_0^l x \varphi_1 dx & 2m(l\omega^2 - \omega^2 \int_0^l \varphi_1^2 dx) \end{vmatrix} = 0$$

For cantilever $\int_0^l \varphi_1^2 dx = l$, $\int_0^l x \varphi_1 dx = \frac{2}{\beta_1^2} = \frac{2l}{3.516} = \frac{l^2}{1.758}$

$$\omega^4 \left[2ml \left(\frac{I_0}{l^2} + \frac{2}{3} ml \right) - \frac{(2ml)^2}{3.08} \right] = \omega^2 \omega_1^2 2ml \left(\frac{I_0}{l^2} + \frac{2}{3} ml \right)$$

$$\omega = \omega_1 \sqrt{\frac{(I_0 + \frac{2}{3} ml^3)}{(I_0 + \frac{2}{3} ml^3) - \frac{2ml^3}{3.090}}}$$

To check for case $I_0 = 0$, the results should be the 2nd mode of a hinged-free beam of length l or 2nd mode of free-free beam of length $2l$.

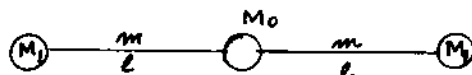
For $I_0 = 0$, $\omega = \omega_1 \sqrt{\frac{3.09}{.090}} = \omega_1 \sqrt{34.5} = 5.85 \omega_1$,

$$= 20.6 \sqrt{\frac{EI}{ml^3}} \quad * \text{ However correct result should be } 15.4 \sqrt{\frac{EI}{ml^3}}$$

* Difficulty due to difference of two numbers in denominator which is small. Needs 2nd mode φ_2 for better results.

9-23 See Prob. 9-21 and 9-22

T has added term $2(\frac{1}{2}) M_1 \dot{y}^2(l, t)$



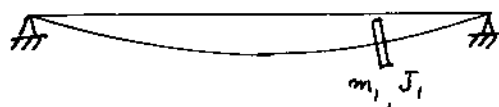
$$= M_1 [\varphi_0 \dot{q}_0 + \varphi_1(l) \dot{q}_1]^2 = M_1 \varphi_0^2 \dot{q}_0^2 + 2M_1 \varphi_0 \varphi_1(l) \dot{q}_0 \dot{q}_1 + M_1 \varphi_1^2(l) \dot{q}_1^2$$

Lagrange's Eq. same as Prob 9-21 + $2M_1 [\ddot{q}_0 + \varphi_1(l) \ddot{q}_1] + 2M_1 [\varphi_1(l) \ddot{q}_0 + \varphi_1^2(l) \ddot{q}_1]$
New freq. eq.

$$\begin{vmatrix} -(M_0 + 2ml + 2M_1)\omega^2 & -(2x.783ml + 2M_1 \varphi_1(l))\omega^2 \\ -(2x.783ml + 2M_1 \varphi_1(l))\omega^2 & [2ml\omega_1^2 - \omega^2(2ml + 2M_1 \varphi_1^2(l))] \end{vmatrix} = 0$$

9-24

The additional mass changes only $T = \frac{1}{2} m_1 \dot{y}^2 + \frac{1}{2} J_1 \dot{y}^2 + \frac{1}{2} \int \dot{y}^2 dm$



$$y(x,t) = \varphi_1(x) q_1(t)$$

$$T = \frac{1}{2} m_1 \varphi_1^2(a) \dot{q}_1^2 + \frac{1}{2} J_1 \varphi_1'^2(a) \dot{q}_1^2 + \frac{1}{2} \int \varphi_1^2(x) dm \cdot \dot{q}_1^2$$

$$\frac{d}{dt} \frac{\partial T}{\partial \dot{q}_1} = m_1 \varphi_1^2(a) \ddot{q}_1 + J_1 \varphi_1'^2(a) \ddot{q}_1 + \ddot{q}_1 \int_0^l \varphi_1^2(x) dx$$

$$= [M_1 + m_1 \varphi_1^2(a) + J_1 \varphi_1'^2(a)] \ddot{q}_1$$

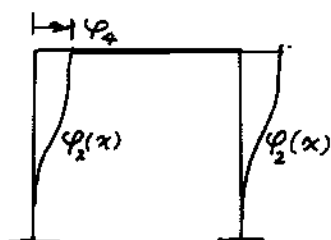
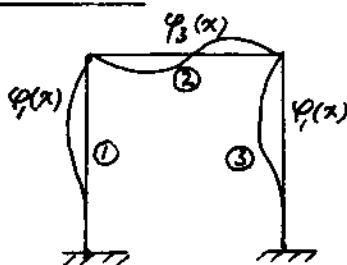
$$= M_1 \left[1 + \frac{m_1}{M_1} \varphi_1^2(a) + \frac{J_1}{M_1} \varphi_1'^2(a) \right] \ddot{q}_1 = M' \ddot{q}_1$$

$$M' \ddot{q}_1 + C_1 \dot{q}_1 + K_1 q_1 = \int_0^l f(x,t) \varphi_1(x) dx$$

$$\therefore \frac{C_1}{M'} = 2\zeta' \omega_1 = \frac{C_1}{M_1 \left[1 + \frac{m_1}{M_1} \varphi_1^2(a) + \frac{J_1}{M_1} \varphi_1'^2(a) \right]}$$

$$\frac{K_1}{M'} = \omega_1'^2 = \frac{K}{\text{same denominator}} \quad \text{etc.}$$

9-25



The modes are available in App. D.

Member ① $w_1(x) = \varphi_1 p_1 + \varphi_2 p_2$

Member ② $w_2(x) = \varphi_3 p_3$

$u_2(x) = 1 p_4$

Member ③ $w_3(x) = \varphi_1 p_5 + \varphi_2 p_6$

So there are 6 coordinates p_1, \dots, p_6

Boundary conditions (4 eqs.)

$$w_1(l) = u_2(0) \quad \varphi_1(l) p_1 + \varphi_2(l) p_2 = p_4$$

$$w_1'(l) = w_2'(0)$$

$$\varphi_1'(l) p_1 + \varphi_2'(l) p_2 = \varphi_3'(0) p_3$$

$$w_2'(l) = w_3'(l)$$

$$\varphi_3'(l) p_3 = \varphi_1'(l) p_5 + \varphi_2'(l) p_6$$

$$u_2(l) = w_3(l)$$

$$p_4 = \varphi_1'(l) p_5 + \varphi_2'(l) p_6$$

9-25 Conti: Gen. mass & gen. stiffness from T & U

$$T = \frac{1}{2} \int_0^{l_1} \dot{w}_1^2 m dx + \frac{1}{2} \int_0^{l_2} [\dot{w}_2^2 + \dot{u}_2^2] m dx + \frac{1}{2} \int_0^{l_1} \dot{w}_3^2 m dx$$

$$U = \frac{1}{2} EI \int_0^{l_1} w_1''^2 dx + \frac{1}{2} EI \int_0^{l_2} w_2''^2 dx + \frac{1}{2} EI \int_0^{l_1} w_3''^2 dx$$

Mass Matrix	$\begin{bmatrix} m_{11} & m_{12} & & & & \\ m_{21} & m_{22} & & & & \\ & & m_{33} & m_{34} & & \\ & & m_{43} & m_{44} & & \\ & & & & m_{55} & m_{56} \\ & & & & m_{65} & m_{66} \end{bmatrix}$	Stiffness Matrix	$\begin{bmatrix} k_{11} & k_{12} & & & & \\ k_{21} & k_{22} & & & & \\ & & k_{33} & k_{34} & & \\ & & k_{43} & k_{44} & & \\ & & & & k_{55} & k_{56} \\ & & & & k_{65} & k_{66} \end{bmatrix}$
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Constraint Eqs.

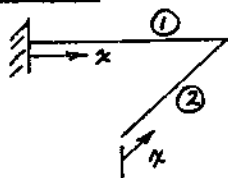
$$\begin{Bmatrix} p_1 \\ p_2 \\ p_3 \\ p_4 \\ p_5 \\ p_6 \end{Bmatrix} = \begin{bmatrix} & & & & & \\ & & & & & \\ & & & & & \\ & & & & & \\ & & & & & \\ & & & & & \end{bmatrix} \begin{Bmatrix} q_1 \\ q_2 \end{Bmatrix}$$

2x6
Matrix

where q_1 & q_2 may
be any of the p_5

System reduces to a 2x2 matrix equation.

9-26



$$\text{Let } w_1(x) = \phi_1 p_1 + \phi_2 p_2 + \phi_3 p_3$$

$$\theta_1(x) = \phi_4 p_4$$

$$w_2(x) = \phi_5 p_5 + \phi_6 p_6 + \phi_7 p_7$$

$$\theta_2(x) = \phi_8 p_8 + \phi_9 p_9$$

Subst. into

$$T = \frac{1}{2} \int \dot{w}^2 dm + \frac{1}{2} \int \frac{J}{A} \dot{\theta}^2 dm$$

$$U = \frac{1}{2} \int EI \left(\frac{d^2 w}{dx^2} \right)^2 dx + \frac{1}{2} \int C \left(\frac{d\theta}{dx} \right)^2 dx$$

to establish m_{ij} and k_{ij}

9-26 Cont:

Constraint eq. at junction of ① and ②

- (1) $w_1(l) = w_2(l)$ defl ① = defl ②
 (2) $\theta_1(l) = -w_2'(l)$ twist ① = bending slope ②
 (3) $C \theta_1'(l) = EI w_2''(l)$ Torque ① = bend. moment ②
 (4) $w_1'''(l) = -w_2'''(l)$ shear ① = shear ②
 (5) $EI w_1''(l) = -C \theta_2'(l)$ bend. mom. ① = torque at ②
 (6) $w_1'(l) = \theta_2(l)$ bend slope ① = twist ②

Mass Matrix

$$\begin{bmatrix}
 m_{11} & m_{12} & m_{13} & & & & & & \\
 m_{21} & m_{22} & m_{23} & & & & & & \\
 m_{31} & m_{32} & m_{33} & & & & & & \\
 & & & m_{44} & & & & & \\
 & & & & 0 & 0 & 0 & 0 & 0 \\
 & & & & m_{55} & m_{56} & m_{57} & 0 & 0 \\
 & & & & m_{65} & m_{66} & m_{67} & 0 & 0 \\
 & & & & m_{75} & m_{76} & m_{77} & 0 & 0 \\
 & & & & & & & m_{88} & m_{89} \\
 & & & & & & & m_{98} & m_{99}
 \end{bmatrix}
 \begin{matrix}
 \\ \\ \\
 \text{all 0} \\
 \\ \\ \\
 \end{matrix}
 \begin{matrix}
 \\ \\ \\
 \text{all 0} \\
 \\ \\ \\
 \end{matrix}
 = 9 \times 9$$

Stiffness Matrix

$$\begin{bmatrix}
 k_{11} & k_{12} & k_{13} & & & & & & \\
 k_{21} & k_{22} & k_{23} & & & & & & \\
 k_{31} & k_{32} & k_{33} & & & & & & \\
 & & & k_{44} & & & & & \\
 & & & & & & & & \\
 & & & & & & & & \\
 & & & & & & k_{77} & & \\
 & & & & & & & & k_{99}
 \end{bmatrix}
 \begin{matrix}
 \\ \\ \\
 \\ \\ \\
 \\ \\
 \end{matrix}
 = 9 \times 9$$

Constraint Matrix = 9×3

$$\begin{bmatrix}
 | \\
 | \\
 | \\
 | \\
 | \\
 | \\
 | \\
 | \\
 |
 \end{bmatrix}$$

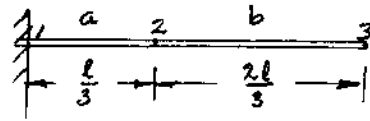
3 gen. coords. any 3 of the P_s

$$\begin{Bmatrix} q_1 \\ q_2 \\ q_3 \end{Bmatrix}$$

Result is

$$\begin{bmatrix} M \\ 3 \times 3 \end{bmatrix}
 \begin{Bmatrix} \ddot{q}_1 \\ \ddot{q}_2 \\ \ddot{q}_3 \end{Bmatrix}
 +
 \begin{bmatrix} K \\ 3 \times 3 \end{bmatrix}
 \begin{Bmatrix} q_1 \\ q_2 \\ q_3 \end{Bmatrix}
 =
 \begin{Bmatrix} 0 \end{Bmatrix}$$

10-1 (see Ex. 10.1-1)



$$M_a = \frac{1}{3} M$$

$$k_a = \frac{EA}{l/3} = \frac{6}{2} \frac{EA}{l}$$

$$M_b = \frac{2}{3} M$$

$$k_b = \frac{EA}{2l/3} = \frac{3}{2} \frac{EA}{l}$$

with $u_1 = 0$

$$\text{Mass Matrix} = \frac{1}{6} \begin{bmatrix} 2(M_a + M_b) & M_b \\ M_b & 2M_b \end{bmatrix} \quad \text{Ex. 10.1-1}$$

$$\text{Stiffness Matrix} = \begin{bmatrix} (k_a + k_b) & -k_b \\ -k_b & k_b \end{bmatrix}$$

Diff. Eq. of Motion with $\ddot{u} = -\omega^2 u$

$$\left\{ \frac{-\omega^2 M}{3 \times 6} \begin{bmatrix} 6 & 2 \\ 2 & 4 \end{bmatrix} + \frac{EA}{2l} \begin{bmatrix} 9 & -3 \\ -3 & 3 \end{bmatrix} \right\} \begin{Bmatrix} u_2 \\ u_3 \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \end{Bmatrix}$$

$$\text{Let } \lambda = \frac{\omega^2 M}{18} \cdot \frac{2l}{EA} = \frac{\omega^2 M l}{9 EA}$$

Characteristic Eq.

$$\begin{vmatrix} (9 - 6\lambda) & -(3 + 2\lambda) \\ -(3 + 2\lambda) & (3 - 4\lambda) \end{vmatrix} = 0 \rightarrow \lambda^2 - 3.30\lambda + .90 = 0$$

$$\lambda = \begin{cases} .30 \\ 3.00 \end{cases}$$

$$\omega^2 = 9 \times \begin{cases} .3 \\ 3.00 \end{cases}$$

$$\omega_1 = 1.643 \sqrt{EA/Ml}$$

$$\omega_2 = 5.196 \sqrt{EA/Ml}$$

$$\text{Exact} \rightarrow \begin{cases} 1.5708 \\ 4.7124 \end{cases}$$

$\therefore 1^{st}$ mode is 4.6% high

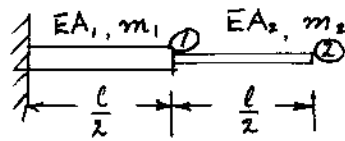
2^{nd} mode is 10.3% high

$$\text{With sta. at } \frac{l}{2}, \quad \lambda^2 - \frac{10}{7}\lambda + \frac{1}{7} = 0$$

$$\omega_1 = 1.6115 \sqrt{\frac{EA}{Ml}} = 1.025 \times \text{Exact} = 2.6\% \text{ high}$$

$$\omega_2 = 5.6293 \sqrt{\frac{EA}{Ml}} = 1.1946 \times " = 19.5\% \text{ high}$$

10-2



$$\begin{aligned} EA_1 &= 2 EA_2 \\ m_1 &= 2 m_2 \\ k_2 &= \frac{2 EA_2}{l} \end{aligned}$$

$$\left[-\frac{\omega^2}{6} \begin{bmatrix} 2(M_a + M_b) & M_b \\ M_b & 2 M_b \end{bmatrix} + \begin{bmatrix} (k_a + k_b) & -k_b \\ -k_b & k_b \end{bmatrix} \right] \begin{Bmatrix} u_1 \\ u_2 \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \end{Bmatrix}$$

$$\left[-\frac{\omega^2 M_2}{6} \begin{bmatrix} 2(2+1) & 1 \\ 1 & 2 \end{bmatrix} + \frac{2 EA_2}{l} \begin{bmatrix} (2+1) & -1 \\ -1 & 1 \end{bmatrix} \right] \begin{Bmatrix} u_1 \\ u_2 \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \end{Bmatrix}$$

$$\text{Let } \lambda = \frac{\omega^2 M_2}{6} \left(\frac{l}{2 EA_2} \right) = \left(\frac{\omega^2 M_2 l}{12 EA_2} \right) \therefore \left| -\lambda \begin{bmatrix} 6 & 1 \\ 1 & 2 \end{bmatrix} + \begin{bmatrix} 3 & -1 \\ -1 & 1 \end{bmatrix} \right| = 0$$

$$\begin{vmatrix} (3-6\lambda) & -(1+\lambda) \\ -(1+\lambda) & (1-2\lambda) \end{vmatrix} = 0$$

$$\lambda^2 - 1.2727\lambda + .1818 = 0$$

$$\lambda_{1,2} = \begin{cases} .1640 \\ 1.1088 \end{cases}$$

$$\omega_1 = 1.4029 \sqrt{\frac{EA_2}{M_2 l}}$$

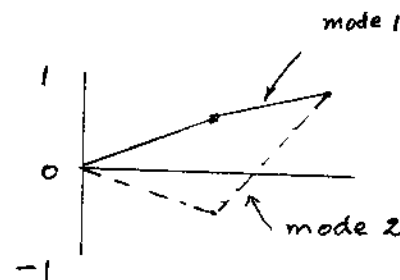
$$\omega_2 = 3.6477 \sqrt{\frac{EA_2}{M_2 l}}$$

$$\text{Mode shapes} \quad (3-6\lambda)u_1 = (1+\lambda)u_2$$

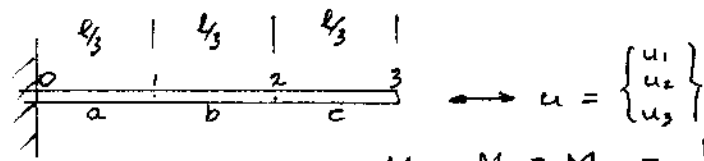
$$\text{or } (1+\lambda)u_1 = (1-2\lambda)u_2$$

$$\left(\frac{u_1}{u_2} \right)_1 = \frac{(1-2\lambda)}{(1+\lambda)} = .5773$$

$$\left(\frac{u_1}{u_2} \right)_2 = -.5258$$



10-3



$$M_a = M_b = M_c = \frac{M}{3}$$

$$k_a = k_b = k_c = \frac{EAa}{l/3} = \frac{3EA}{l}$$

Mass matrix

$$\begin{bmatrix} \frac{M_a}{6} \begin{bmatrix} 2 & 1 \\ 1 & 2+2 \end{bmatrix} & & \\ & \frac{M_b}{6} \begin{bmatrix} 2+2 & 1 \\ 1 & 2 \end{bmatrix} & \\ & & \frac{M_c}{6} \begin{bmatrix} 2+2 & 1 \\ 1 & 2 \end{bmatrix} \end{bmatrix}$$

Stiffness matrix

$$k_a \begin{bmatrix} \begin{bmatrix} 1 & -1 \\ -1 & 1+1 \end{bmatrix} & & \\ & \begin{bmatrix} 1+1 & -1 \\ -1 & 1+1 \end{bmatrix} & \\ & & \begin{bmatrix} 1+1 & -1 \\ -1 & 1 \end{bmatrix} \end{bmatrix}$$

$$\frac{M_a}{6} \begin{bmatrix} 2 & 1 & 0 & 0 \\ 1 & 4 & 1 & 0 \\ 0 & 1 & 4 & 1 \\ 0 & 0 & 1 & 2 \end{bmatrix} \begin{Bmatrix} \ddot{u}_0 \\ \ddot{u}_1 \\ \ddot{u}_2 \\ \ddot{u}_3 \end{Bmatrix} + k_a \begin{bmatrix} 1 & -1 & 0 & 0 \\ -1 & 2 & -1 & 0 \\ 0 & -1 & 2 & -1 \\ 0 & 0 & -1 & 1 \end{bmatrix} \begin{Bmatrix} u_0 \\ u_1 \\ u_2 \\ u_3 \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \\ 0 \\ 0 \end{Bmatrix}$$

$u_0 = 0$ \therefore cross out as above

$$\frac{M}{18} \begin{bmatrix} 4 & 1 & 0 \\ 1 & 4 & 1 \\ 0 & 1 & 2 \end{bmatrix} \begin{Bmatrix} \ddot{u}_1 \\ \ddot{u}_2 \\ \ddot{u}_3 \end{Bmatrix} + \frac{3EA}{l} \begin{bmatrix} 2 & -1 & 0 \\ -1 & 2 & -1 \\ 0 & -1 & 1 \end{bmatrix} \begin{Bmatrix} u_1 \\ u_2 \\ u_3 \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \\ 0 \end{Bmatrix}$$

Computer solution: $\lambda = \frac{Ml\omega^2}{54EA}$

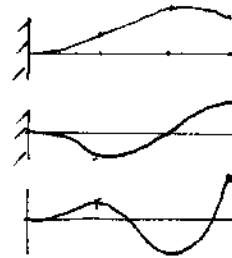
$$\omega_1 = 1.522 \sqrt{\frac{EA}{Ml}} \quad \left(\frac{EA}{Ml} = \frac{1}{l} \sqrt{\frac{E}{\rho}} \right) \quad \text{Exact } 1.571 \sqrt{\frac{E}{\rho}} \quad \therefore 3\% \text{ low}$$

$$\omega_2 = 5.1545 \quad \text{"} \quad 4.712 \quad \text{"} \quad 9\% \text{ high}$$

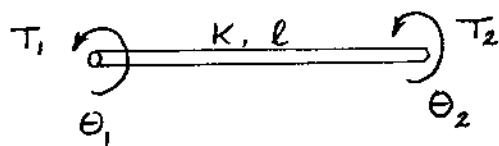
$$\omega_3 = 9.4329 \quad \text{"} \quad 7.85 \quad \text{"} \quad 20\% \text{ high}$$

Mode Shapes

$$\begin{Bmatrix} 0 \\ .415 \\ .728 \\ .544 \end{Bmatrix}_1 \quad \begin{Bmatrix} 0 \\ -.811 \\ .0268 \\ .583 \end{Bmatrix}_2 \quad \begin{Bmatrix} 0 \\ .410 \\ -.684 \\ .602 \end{Bmatrix}_3$$



10 - 4 With linear twist, the problem is identical to that of the longitudinal vibration.
Torsional element of length l

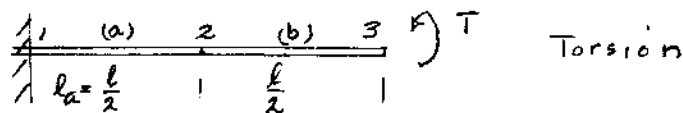


Mass = $\rho A l$ Stiffness = $\frac{I_p G}{l}$ of length l

$J_p = \frac{MR^2}{2}$ = mass polar moment of inertia

Mass matrix = $\frac{\rho A l}{6} \begin{bmatrix} 2 & 1 \\ 1 & 2 \end{bmatrix}$ Stiffness = $\frac{I_p G}{l} \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix}$

10 - 5



$M_a \frac{R^2}{2} = J_a$

Let J = mass polar moment of inertia for entire rod

Then $J_a = \frac{1}{2} J$, $K_a = \frac{I_p G}{l_a} = \frac{2 I_p G}{l}$

$$\frac{J_a}{6} \begin{bmatrix} 2 & 1 & 0 \\ 1 & 4 & 1 \\ 0 & 1 & 2 \end{bmatrix} \begin{Bmatrix} \ddot{\theta}_1 \\ \ddot{\theta}_2 \\ \ddot{\theta}_3 \end{Bmatrix} + K_a \begin{bmatrix} 1 & -1 & 0 \\ -1 & 2 & -1 \\ 0 & -1 & 1 \end{bmatrix} \begin{Bmatrix} \theta_1 \\ \theta_2 \\ \theta_3 \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \\ 0 \end{Bmatrix}$$

Units: J = mass in^2 , $I_p = \text{in}^4$, $G = \text{lb/in}^2$, $l = \text{in}$.

Let $\lambda = \frac{\omega^2 J_a}{6 K_a}$ $J_a = \frac{J}{2}$, $K_a = 2 K$

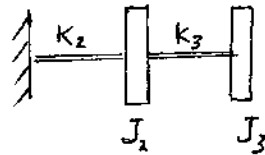
Same prob. as longitudinal vibr.

$$\omega = \begin{cases} 1.6114 \sqrt{\frac{K}{J}} \\ 5.629 \text{ "} \end{cases} \quad \text{where } K = \frac{I_p G}{l}, \quad J = M \frac{R^2}{2}$$

$$\frac{K}{J} = \frac{2 I_p G}{M l R^2} = \frac{G A}{M l}$$

10-6

lumped mass torsional system

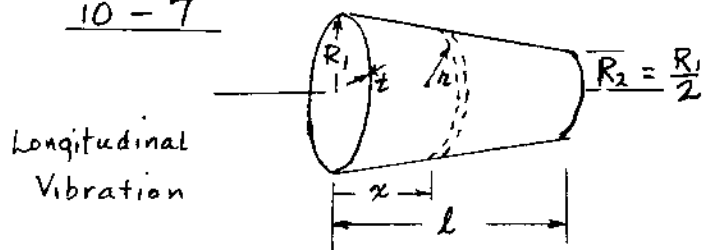


$$\begin{bmatrix} J_2 & 0 \\ 0 & J_3 \end{bmatrix} \begin{Bmatrix} \ddot{\theta}_2 \\ \ddot{\theta}_3 \end{Bmatrix} + \begin{bmatrix} (K_2 + K_3) & -K_3 \\ -K_3 & K_3 \end{bmatrix} \begin{Bmatrix} \theta_2 \\ \theta_3 \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \end{Bmatrix}$$

$$\begin{bmatrix} [(K_2 + K_3) - \omega^2 J_2] & -K_3 \\ -K_3 & [K_3 - \omega^2 J_3] \end{bmatrix} \begin{Bmatrix} \theta_2 \\ \theta_3 \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \end{Bmatrix} \quad \therefore \text{same form}$$

The problem here is assigning proper values of J_2 and J_3 , to be equivalent to distributed system.

10-7



$$\xi = x/l$$

$$\varphi_1 = (1 - \xi)$$

$$\varphi_2 = \xi$$

$$u = u_1(1 - \xi) + u_2 \xi$$

$$m_{ij} = \int \varphi_i \varphi_j dm$$

$$r = R_1(1 - \frac{1}{2}\xi)$$

$$dm = \rho \cdot 2\pi r t \cdot dx, \quad \rho = \text{mass density}$$

$$A = 2\pi r t$$

$$\therefore m_{11} = \rho \cdot 2\pi t R_1 \int_0^1 (1 - \frac{1}{2}\xi)(1 - \xi)^2 l d\xi = m_1 \int_0^1 (1 - \frac{5}{2}\xi + 2\xi^2 - \frac{1}{2}\xi^3) d\xi$$

$$\text{where } m_1 = \rho \cdot 2\pi t R_1 l = \rho A_1 l$$

$$m_{11} = m_1 \left(1 - \frac{5}{4} + \frac{2}{3} - \frac{1}{8} \right) = \frac{m_1}{24} (24 - 30 + 16 - 3) = \frac{7}{24} m_1$$

$$A_1 = 2\pi t R_1$$

10-7 Cont.

$$m_{11} = m_1 \int_0^1 (1 - \frac{1}{2}\xi)(1 - \xi)\xi d\xi = \frac{3}{24} m_1$$

$$m_{22} = m_1 \int_0^1 (1 - \frac{1}{2}\xi)\xi^2 d\xi = \frac{5}{24} m_1$$

$$\therefore m = \frac{m_1}{24} \begin{bmatrix} 7 & 3 \\ 3 & 5 \end{bmatrix}$$

$$k_{ij} = \int EA \varphi_i' \varphi_j' l d\xi \quad \varphi_1' = \frac{d\varphi_1}{dx} = -\frac{1}{l}, \quad \varphi_2' = \frac{1}{l}$$

$$k_{11} = E \frac{2\pi t R_1}{l^2} \int_0^1 (1 - \frac{1}{2}\xi) l d\xi = \frac{EA_1}{l} (1 - \frac{1}{4}) = \frac{3}{4} \frac{EA_1}{l}$$

$$\text{where } A_1 = 2\pi t R_1$$

$$k_{12} = -\frac{3}{4} \frac{EA_1}{l} = k_{21}, \quad k_{22} = \frac{3}{4} \frac{EA_1}{l}$$

$$\therefore k = \frac{3}{4} \frac{EA_1}{l} \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix}$$

Eq. of motion

$$\frac{\rho A_1 l}{24} \begin{bmatrix} 7 & 3 \\ 3 & 5 \end{bmatrix} \begin{Bmatrix} \ddot{u}_1 \\ \ddot{u}_2 \end{Bmatrix} + \frac{3}{4} \frac{EA_1}{l} \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix} \begin{Bmatrix} u_1 \\ u_2 \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \end{Bmatrix}$$

$$\text{Let } \lambda = \frac{\omega^2 \rho l^2}{18E}$$

$$\begin{vmatrix} (1-7\lambda) & -(1+3\lambda) \\ -(1+3\lambda) & (1-5\lambda) \end{vmatrix} = 0 \quad \therefore \lambda (26\lambda - 6) = 0$$

$$\lambda_0 = 0$$

$$\lambda_1 = \frac{6}{26} = \frac{\omega^2 \rho l^2}{18E} = \frac{\omega^2 (\rho A_1 l) l}{18EA_1} = \frac{\omega^2 m_1 l}{18EA_1}$$

10-7 Cont.

$$\omega_1^2 = \left(\frac{6}{26} \times 18\right) \left(\frac{E}{\rho l^2}\right) = 4.1538 \left(\frac{E}{\rho l^2}\right)$$

$$\omega_1 = 2.038 \sqrt{\frac{EA_1}{m_1 l}} = \underline{\underline{2.038 \sqrt{\frac{E}{\rho l^2}}}} \quad \begin{array}{l} \text{for 1-element} \\ \text{solution} \\ \text{for Longit. vib.} \end{array}$$

where A_1 = cross section area at ①

$M_1 = m_1 l$ = mass of entire uniform cylinder of radius R_1 and length l .

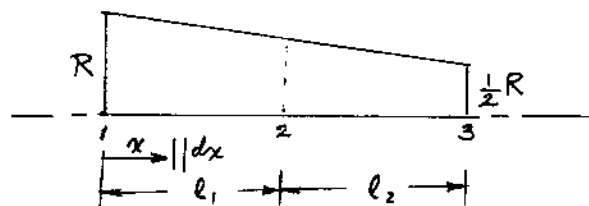
If we have a uniform cylindrical shell of Radius R_1 and length l , with one end fixed with other end free, treated like a helical spring with effective mass $\frac{m_1}{3}$ at free end, the natural frequency will be

$$\omega_1 = \sqrt{\frac{EA_1}{(\frac{m_1}{3})l}} = 1.732 \sqrt{\frac{EA_1}{m_1 l}}$$

10-8 Same conical tube as in 10-7 treated as two elements of equal length, 1-2 and 2-3.

$$r_{1-2} = R \left(1 - \frac{1}{4} \frac{x}{l_1}\right)$$

$$\xi = \frac{x}{l_1}$$



$$m_{ij} = \int \varphi_i \varphi_j dm = \int \varphi_i \varphi_j 2\pi \rho t r_{1-2} dx = 2\pi \rho t R \int_0^1 \left(1 - \frac{1}{4}\xi\right) \varphi_i \varphi_j l_1 d\xi$$

$$m_{11} = 2\pi \rho t R l_1 \int_0^1 \left(1 - \frac{1}{4}\xi\right) (1-\xi)^2 d\xi = 2\pi \rho t R l_1 (.3125)$$

$$m_{12} = \dots \int_0^1 \left(1 - \frac{1}{4}\xi\right) (1-\xi) \xi d\xi = \dots (-.1458)$$

$$m_{22} = \dots \int_0^1 \left(1 - \frac{1}{4}\xi\right) \xi^2 d\xi = \dots (.2708)$$

10-8 Cont.

$$m = 2\pi \rho t R l_1 \begin{bmatrix} .3125 & .1458 \\ .1458 & .2708 \end{bmatrix}$$

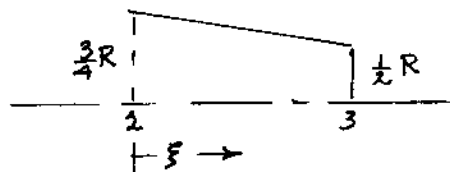
$$k_{ij} = \int_0^l EA \varphi_i' \varphi_j' l_1 d\xi \quad \therefore k_{11} = \int_0^l 2\pi R E \left(1 - \frac{1}{4}\xi\right) \left(-\frac{1}{l_1}\right)^2 l_1 d\xi$$

$$= \frac{2\pi R E}{l_1} (.8750)$$

$$k_{12} = -k_{11}, \quad k_{22} = k_{11}$$

$$k = \frac{2\pi R E}{l_1} \begin{bmatrix} .8750 & -.8750 \\ -.8750 & .8750 \end{bmatrix}$$

Element 2-3



$$r = \frac{3}{4} R \left(1 - \frac{1}{3}\xi\right)$$

$$A = 2\pi t r = 2\pi t R \frac{3}{4} \left(1 - \frac{1}{3}\xi\right)$$

$$dm = 2\pi \rho t r dx = 2\pi \rho t R \left(1 - \frac{1}{3}\xi\right) l_1 d\xi$$

$$m_{11} = \frac{3}{2} \pi \rho t R l_1 \int_0^1 \left(1 - \frac{1}{3}\xi\right) \left(1 - \xi\right)^2 d\xi = \frac{3}{2} \pi \rho t R l_1 (.3056)$$

$$m_{12} = \frac{3}{2} \pi \rho t R l_1 (.1389), \quad m_{22} = \frac{3}{2} \pi \rho t R l_1 (.2500)$$

$$m = \frac{3}{2} \pi \rho t R l_1 \begin{bmatrix} .3056 & .1389 \\ .1389 & .2500 \end{bmatrix} = 2\pi \rho t R l_1 \begin{bmatrix} .2292 & .1042 \\ .1042 & .1875 \end{bmatrix}$$

$$k_{11} = \frac{3}{2} \pi \rho t \frac{R}{l_1} \int_0^1 \left(1 - \frac{1}{3}\xi\right) d\xi = \frac{3}{2} \pi \rho t \frac{R}{l_1} \left(\frac{5}{6}\right)$$

$$k_{12} = -k_{11}, \quad k_{22} = k_{11}$$

$$\therefore k = \frac{3}{2} \pi \times \frac{5}{6} \rho t \frac{R}{l_1} \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix}$$

10-8 Cont.

Assemble element matrices

$$m = 2\pi \rho t R l_1 \begin{bmatrix} .3125 & .1458 & 0 \\ .1458 & (.2708 + 2292) & .1042 \\ 0 & .1042 & .1875 \end{bmatrix}$$

$$k = \frac{\pi R E t}{l_1} \begin{bmatrix} 1.75 & -1.75 & 0 \\ -1.75 & (1.75 + 1.25) & -1.25 \\ 0 & -1.25 & 1.25 \end{bmatrix} \quad \lambda = \frac{\omega^2 \rho l_1}{E}$$

With $u_1 = 0$, eqs. of motion leads to chara. eq.

$$\left| -\lambda \begin{bmatrix} .5000 & .1042 \\ .1042 & .1875 \end{bmatrix} + \begin{bmatrix} 3.00 & -1.25 \\ -1.25 & 1.25 \end{bmatrix} \right| = 0$$

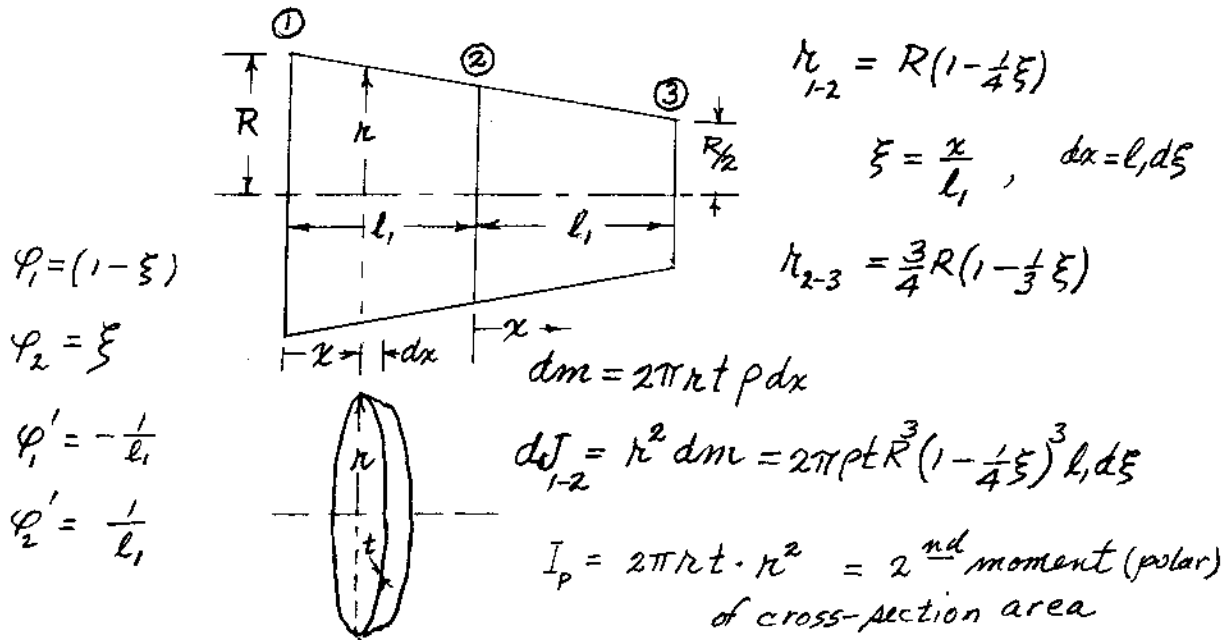
$$\begin{vmatrix} (3.00 - .500\lambda) & -(1.25 + .1042\lambda) \\ -(1.25 + .1042\lambda) & (1.25 - .1875\lambda) \end{vmatrix} = 0$$

$$\lambda^2 - 17.46 \lambda + 26.380 = 0$$

$$\begin{aligned} \lambda_1 &= 1.6708 & \omega_1 &= 1.2926 \sqrt{\frac{E}{\rho l_1^2}} \\ \lambda_2 &= 15.7892 & \omega_2 &= 3.9736 \sqrt{\frac{E}{\rho l_1^2}} \end{aligned} \quad l_1 = \frac{l}{2}$$

$$\therefore \left. \begin{aligned} \omega_1 &= 2.585 \sqrt{\frac{E}{\rho l^2}} \\ \omega_2 &= 7.947 \sqrt{\frac{E}{\rho l^2}} \end{aligned} \right\} \text{longitudinal vibr (2 elements)}$$

10 - 9(a) Torsional Vibration of Conical Shell.



Stiffness of element 1-2.

$$k_{ij} = G \int_0^{l_1} I_p \varphi_i' \varphi_j' dx = G \cdot 2\pi t R^3 \int_0^1 (1 - \frac{1}{4}\xi)^3 \varphi_i' \varphi_j' l_1 d\xi$$

$$\begin{aligned}
 k_{11} &= G \cdot 2\pi t R^3 \int_0^1 (1 - \frac{1}{4}\xi)^3 (\frac{1}{l_1}) l_1 d\xi = \frac{G 2\pi t R^3}{l_1} \int_0^1 (1 - \frac{3}{4}\xi + \frac{3}{16}\xi^2 - \frac{1}{64}\xi^3) d\xi \\
 &= \frac{G 2\pi t R^3}{l_1} (1 - \frac{3}{8} + \frac{1}{16} - \frac{1}{256}) = \frac{G 2\pi t R^3}{l_1} (.6836)
 \end{aligned}$$

$$k_{12} = k_{21} = -k_{11}, \quad k_{22} = k_{11}$$

$$\therefore K_{1-2} = \frac{G 2\pi t R^3}{l_1} (.6836) \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix}$$

Stiffness of element 2-3

$$\begin{aligned}
 k_{11} &= \frac{G 2\pi t (.75R)^3}{l_1} \int_0^1 (1 - \frac{1}{3}\xi)^3 d\xi = \frac{G 2\pi t R^3}{l_1} (.4219) \int_0^1 (1 - \xi + \frac{1}{3}\xi^2 - \frac{1}{27}\xi^3) d\xi \\
 &= \frac{G 2\pi t R^3}{l_1} (.4219)(.6019) = \frac{G 2\pi t R^3}{l_1} (.2539)
 \end{aligned}$$

$$K_{2-3} = \frac{G 2\pi t R^3}{l_1} (.2539) \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix} = \frac{G 2\pi t R^3}{l_1} (.6836) \begin{bmatrix} .3714 & -.3714 \\ -.3714 & .3714 \end{bmatrix}$$

10-9 (a) Cont.

$$\text{Combined stiffness} = \frac{G 2\pi t R^3}{l/2} (1.6836) \begin{bmatrix} 1 & -1 \\ -1 & 1.3714 & -.3714 \\ & -.3714 & .3714 \end{bmatrix}$$

For $\theta_1 = 0$

$$K\theta = (1.3672) \frac{G 2\pi t R^3}{L} \begin{bmatrix} 1.3714 & -.3714 \\ -.3714 & .3714 \end{bmatrix} \begin{Bmatrix} \theta_2 \\ \theta_3 \end{Bmatrix}$$

Mass matrix, element 1-2

$$J_{ij} = \int_0^1 \varphi_i \varphi_j dJ = 2\pi \rho t R^3 \int_0^1 \left(1 - \frac{1}{4}\xi\right)^3 \varphi_i \varphi_j l, d\xi$$

$$\begin{aligned} J_{11} &= 2\pi \rho t R^3 \int_0^1 \left(1 - \frac{1}{4}\xi\right)^3 (1 - \xi)^2 l, d\xi \\ &= 2\pi \rho t R^3 l, \int_0^1 \left(1 - \frac{11}{4}\xi + \frac{43}{16}\xi^2 - \frac{73}{64}\xi^3 + \frac{7}{32}\xi^4 - \frac{1}{64}\xi^5\right) d\xi \\ &= \quad \quad \quad \left(1 - \frac{11}{8} + \frac{43}{48} - \frac{73}{256} + \frac{7}{160} - \frac{1}{384}\right) \\ &= 2\pi \rho t R^3 l, (.2768) \end{aligned}$$

$$\begin{aligned} J_{12} &= 2\pi \rho t R^3 l, \int_0^1 \left(1 - \frac{1}{4}\xi\right)^3 (1 - \xi) \xi d\xi \\ &\quad \quad \quad \int_0^1 \left(\xi - \frac{7}{4}\xi^2 + \frac{15}{16}\xi^3 - \frac{13}{64}\xi^4 + \frac{1}{64}\xi^5\right) d\xi \\ &= 2\pi \rho t R^3 l, (.1130) \end{aligned}$$

$$\begin{aligned} J_{22} &= 2\pi \rho t R^3 l, \int_0^1 \left(1 - \frac{1}{4}\xi\right)^3 \xi^2 d\xi = \int_0^1 \left(\xi^2 - \frac{3}{4}\xi^3 + \frac{3}{16}\xi^4 - \frac{1}{64}\xi^5\right) d\xi () \\ &= 2\pi \rho t R^3 l, \left(\frac{1}{3} - \frac{3}{16} + \frac{3}{80} - \frac{1}{384}\right) = 2\pi \rho t R^3 l, (.1804) \end{aligned}$$

10-9(a) Cont.

$$J_{1-2} = 2\pi\rho t R^3 l_1 \begin{bmatrix} .2768 & .1130 \\ .1130 & .1804 \end{bmatrix} \quad (l_1 = \frac{l}{2})$$

Mass matrix, element 2-3 $h_{2-3} = \frac{3}{4}R(1 - \frac{1}{3}\xi)$

$$dJ_{2-3} = 2\pi h t \rho \cdot h^2 = 2\pi\rho t \left(\frac{3}{4}R\right)^3 \left(1 - \frac{1}{3}\xi\right)^3$$

$$J_{22} = 2\pi\rho t R^3 (.4219) \int_0^1 \left(1 - \frac{1}{3}\xi\right)^3 (1 - \xi)^2 l_1 d\xi$$

$$\therefore l_1 \int_0^1 \left(1 - 3\xi + \frac{10}{3}\xi^2 - 1.7036\xi^3 + .40737\xi^4 - .03704\xi^5\right) d\xi$$

$$= 2\pi\rho t R^3 l_1 (.4219) (.2605) = 2\pi\rho t R^3 l_1 (.1099)$$

$$J_{23} = \text{same term} \times l_1 \int_0^1 \left(1 - \frac{1}{3}\xi\right)^3 (1 - \xi) \xi d\xi$$

$$= \text{"} \int_0^1 \left(\xi - 2\xi^2 + \frac{4}{3}\xi^3 - \frac{10}{27}\xi^4 + \frac{1}{27}\xi^5\right) d\xi$$

$$= 2\pi\rho t R^3 l_1 (.4219) (.0988) = 2\pi\rho t R^3 l_1 (.0417)$$

$$J_{33} = \text{same term} \times l_1 \int_0^1 \left(1 - \frac{1}{3}\xi\right)^3 \xi^2 d\xi = 2\pi\rho t R^3 l_1 (.4219) (.1438)$$

$$= 2\pi\rho t R^3 l_1 (.0607)$$

Combined mass term with $\theta_1 = 0$

$$2\pi\rho t R^3 l_1 \begin{bmatrix} (.1099 + .1804) & .0417 \\ .0417 & .0607 \end{bmatrix} \begin{Bmatrix} \ddot{\theta}_2 \\ \ddot{\theta}_3 \end{Bmatrix} \quad (l_1 = \frac{l}{2})$$

$$J = \pi\rho t R^3 l \begin{bmatrix} .2903 & .0417 \\ .0417 & .0607 \end{bmatrix}$$

10-9(a) Cont.

Eq. of motion: Let $\lambda = \frac{\omega^2 \cancel{\rho} \cancel{R} \cancel{l}}{1.3672 G \cancel{2} \cancel{\rho} \cancel{R} \cancel{l}}$

$$\lambda = \frac{\omega^2 \rho l^2}{2.734 G}$$

$$\left[-\lambda \begin{bmatrix} .2903 & .0417 \\ .0417 & .0607 \end{bmatrix} + \begin{bmatrix} 1.3714 & -.3714 \\ -.3714 & .3714 \end{bmatrix} \right] \begin{Bmatrix} \theta_2 \\ \theta_3 \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \end{Bmatrix}$$

$$\begin{vmatrix} (1.3714 - .2903\lambda) & -(.3714 + .0417\lambda) \\ -(.3714 - .0417\lambda) & (.3714 - .0607\lambda) \end{vmatrix} = 0$$

$$\lambda^2 - 10.069\lambda + 23.358 = 0$$

$$\lambda_1 = 3.624$$

$$\lambda_2 = 6.438$$

$$\omega_1^2 = 9.908 \frac{G}{\rho l^2}$$

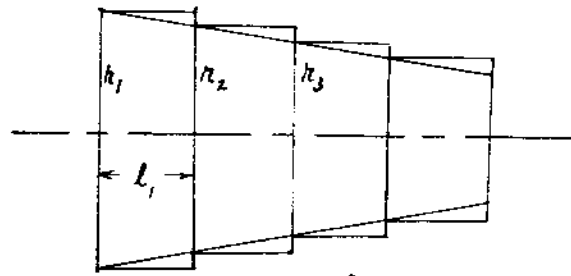
$$\omega_2^2 = 17.603 \frac{G}{\rho l^2}$$

$$\underline{\underline{\omega_1 = 3.147 \sqrt{\frac{G}{\rho l^2}}}}$$

$$\underline{\underline{\omega_2 = 4.196 \sqrt{\frac{G}{\rho l^2}}}}$$

Results unchecked.

10-9(b)



Torsion problem.

Assume short elements \therefore constant radius for each section.

Element stiffness:

$$A_i = 2\pi r_i t, \quad I_{p_i} = A r_i^2, \quad G I_{p_i} = G A_i r_i^2$$

$$K_i = \frac{G I_{p_i}}{l_i} = \frac{G 2\pi t r_i^3}{l_i}$$

Assume straight line variation in twist $\therefore \varphi_{1_i}, \varphi_{2_i}$ applies to each element $\varphi_{1_i} = (1-\xi), \varphi_{2_i} = \xi$

where $\xi = \frac{x}{l_i}$. Then $K_i = \frac{G I_{p_i}}{l_i} \int_0^1 \varphi_{1_i}' \varphi_{2_i}' l_i d\xi$

$$\int_0^1 \varphi_{1_i}' \varphi_{2_i}' l_i d\xi = \pm 1$$

$$K \text{ for element} = G \left(\frac{2\pi t r_i^3}{l_i} \right) \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix} \quad l_i = \frac{l}{n}$$

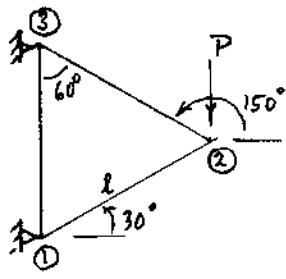
Element mass:

$$J \text{ for element} = \frac{2\pi t \rho l_i r_i^3}{6} \begin{bmatrix} 2 & 1 \\ 1 & 2 \end{bmatrix}$$

N section eq.

$$\rho \frac{2\pi t l}{6n} \begin{bmatrix} 2r_1^3 & r_1^3 \\ r_1^3 & 2(r_1^3 + r_2^3) \\ r_2^3 & 2(r_2^3 + r_3^3) \\ r_3^3 & 2(r_3^3 + r_4^3) \\ \vdots & \vdots \end{bmatrix} \begin{Bmatrix} \ddot{\theta}_1 \\ \ddot{\theta}_2 \\ \ddot{\theta}_3 \\ \ddot{\theta}_4 \\ \vdots \end{Bmatrix} + \frac{G 2\pi t m}{l} \begin{bmatrix} r_1^3 & -r_1^3 \\ -r_1^3 & (r_1^3 + r_2^3) - r_2^3 \\ & -r_2^3 & (r_2^3 + r_3^3) \\ & & \ddots \end{bmatrix} \begin{Bmatrix} \theta_1 \\ \theta_2 \\ \theta_3 \\ \vdots \end{Bmatrix}$$

10-10



Element 1-2 $\alpha = 30^\circ$ $C = .866$, $S = .500$
 $C^2 = .750$ $S^2 = .250$ $CS = .433$

$$\bar{k}_{12} = \frac{EA}{l} \begin{bmatrix} \dots & \dots \\ \dots & \dots \\ \dots & \dots \\ \dots & \dots \end{bmatrix} \begin{Bmatrix} \bar{u}_1 = 0 \\ \bar{v}_1 = 0 \\ \bar{u}_2 \\ \bar{v}_2 \end{Bmatrix}$$

Element 2-3, $\alpha = 150^\circ$ $C = -.866$ $S = .50$

$$\bar{k}_{23} = \frac{EA}{l} \begin{bmatrix} .750 & -.433 & -.750 & .433 \\ -.433 & .250 & .433 & -.250 \\ -.750 & .433 & .750 & -.433 \\ .433 & -.250 & -.433 & .250 \end{bmatrix} \begin{Bmatrix} \bar{u}_2 \\ \bar{v}_2 \\ \bar{u}_3 \\ \bar{v}_3 \end{Bmatrix}$$

Assemble

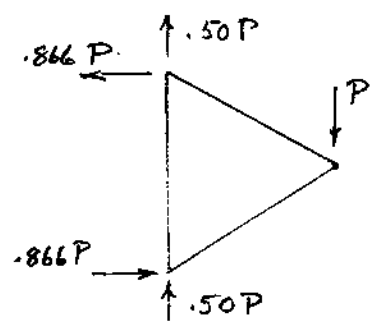
$$\begin{Bmatrix} F_{2x} = 0 \\ F_{2y} = -P \\ F_{3x} \\ F_{3y} \end{Bmatrix} = \frac{EA}{l} \begin{bmatrix} 1.50 & 0 & -.75 & .433 \\ 0 & .50 & .433 & -.25 \\ -.75 & .433 & .75 & -.433 \\ .433 & -.25 & -.433 & .25 \end{bmatrix} \begin{Bmatrix} \bar{u}_2 \\ \bar{v}_2 \\ \bar{u}_3 = 0 \\ \bar{v}_3 = 0 \end{Bmatrix}$$

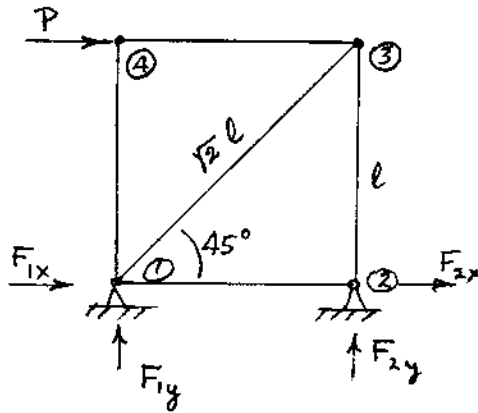
$$F_{2x} = 0 = \frac{EA}{l} 1.50 \bar{u}_2 \quad \therefore \bar{u}_2 = 0$$

$$F_{2y} = -P = \frac{EA}{l} .50 \bar{v}_2 \quad \therefore \bar{v}_2 = -\frac{2Pl}{EA}$$

$$F_{3x} = \frac{EA}{l} .433 \bar{v}_2 = \frac{EA}{l} .433 \left(-\frac{2Pl}{EA} \right) = -.866P$$

$$F_{3y} = -.25 \bar{v}_2 \frac{EA}{l} = -.25 \frac{EA}{l} \left(-\frac{2Pl}{EA} \right) = 0.50P$$





$$\bar{k}_{12} = \frac{EA}{l} \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix} \quad \bar{m}_{12} = \frac{ml}{6} \begin{bmatrix} 2 & 1 \\ 1 & 2 \end{bmatrix}$$

Element 1-2, $\alpha=0$, $s=0$, $c=1$

$$\bar{k}_{1-2} = \frac{EA}{l} \begin{bmatrix} 1 & 0 & -1 & 0 \\ 0 & 0 & 0 & 0 \\ -1 & 0 & 1 & 0 \\ 0 & 0 & 0 & 0 \end{bmatrix} \begin{matrix} 1 \\ 2 \end{matrix}$$

Element 2-3 $\alpha=90^\circ$, $s=1$, $c=0$

$$\bar{k}_{2-3} = \frac{EA}{l} \begin{bmatrix} 0 & 0 & 0 & 0 \\ 0 & 1 & 0 & -1 \\ 0 & 0 & 0 & 0 \\ 0 & -1 & 0 & 1 \end{bmatrix} \begin{matrix} 2 \\ 3 \end{matrix}$$

Element 3-4 $\alpha=180^\circ$, $s=0$, $c=-1$

$$\bar{k}_{3-4} = \frac{EA}{l} \begin{bmatrix} 1 & 0 & -1 & 0 \\ 0 & 0 & 0 & 0 \\ -1 & 0 & 1 & 0 \\ 0 & 0 & 0 & 0 \end{bmatrix} \begin{matrix} 3 \\ 4 \end{matrix}$$

Element 4-1 $\alpha=270^\circ$, $s=-1$, $c=0$

$$\bar{k}_{4-1} = \frac{EA}{l} \begin{bmatrix} 0 & 0 & 0 & 0 \\ 0 & 1 & 0 & -1 \\ 0 & 0 & 0 & 0 \\ 0 & -1 & 0 & 1 \end{bmatrix} \begin{matrix} 4 \\ 1 \end{matrix}$$

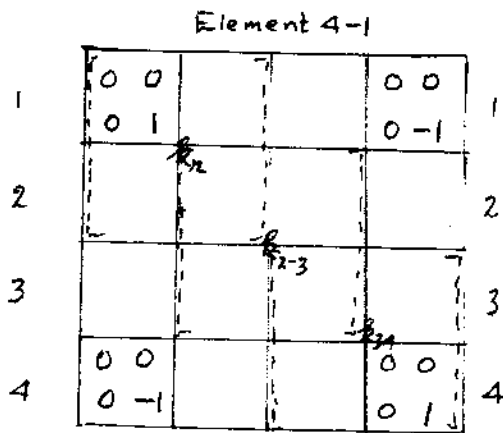
Element 1-3 $\alpha=45^\circ$, $s = \frac{\sqrt{2}}{2}$, $c = \frac{\sqrt{2}}{2}$ Length = $\sqrt{2}l$

$$\bar{k}_{13} = \frac{EA}{\sqrt{2}l} \begin{bmatrix} .5 & .5 & -.5 & -.5 \\ .5 & .5 & -.5 & -.5 \\ -.5 & -.5 & .5 & .5 \\ -.5 & -.5 & .5 & .5 \end{bmatrix} \begin{matrix} 1 \\ 3 \end{matrix}$$

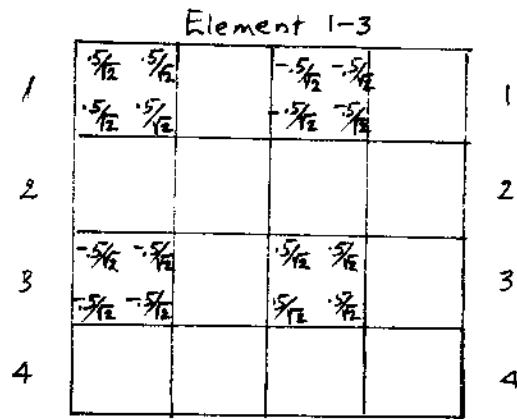
$$\begin{Bmatrix} \bar{F}_{1x} \\ \bar{F}_{1y} \\ \bar{F}_{2x} \\ \bar{F}_{2y} \\ \bar{F}_{3x} \\ \bar{F}_{3y} \\ \bar{F}_{4x} \\ \bar{F}_{4y} \end{Bmatrix} = [K] \begin{Bmatrix} \bar{u}_1 = 0 \\ \bar{v}_1 = 0 \\ \bar{u}_2 = 0 \\ \bar{v}_2 = 0 \\ \bar{u}_3 \\ \bar{v}_3 \\ \bar{u}_4 \\ \bar{v}_4 \end{Bmatrix}$$

Since all displacements for ① and ② are zero, we need only the lower part of this matrix or the lower right section of K . However for instructional purposes we will fill in the elements into the 8×8 space for K .

There is no problem with placement of \bar{K}_{12} , \bar{K}_{23} and \bar{K}_{34} . Element \bar{K}_{4-1} and \bar{K}_{13} must be placed as follows:



dotted squares indicate placement of \bar{K}_{12} , \bar{K}_{23} , \bar{K}_{34}



superimpose this onto diagram on left.

Considering only those terms in lower right quarter, we have

$$\begin{Bmatrix} \bar{F}_{3x}=0 \\ \bar{F}_{3y}=0 \\ \bar{F}_{4x}=P \\ \bar{F}_{4y}=0 \end{Bmatrix} = \frac{EA}{L} \begin{bmatrix} (1+\frac{5}{12}) & \frac{5}{12} & -1 & 0 \\ \frac{5}{12} & (1+\frac{5}{12}) & 0 & 0 \\ -1 & 0 & 1 & 0 \\ 0 & 0 & 0 & 1 \end{bmatrix} \begin{Bmatrix} \bar{u}_3 \\ \bar{v}_3 \\ \bar{u}_4 \\ \bar{v}_4 \end{Bmatrix}$$

10-11 Cont. The last matrix give

$$(1) \quad 0 = \frac{EA}{l} \left[(1 + \frac{5}{12}) \bar{u}_3 + \frac{5}{12} \bar{v}_3 - \bar{u}_4 \right]$$

$$(2) \quad 0 = \frac{EA}{l} \left[\frac{5}{12} \bar{u}_3 + (1 + \frac{5}{12}) \bar{v}_3 \right] \rightarrow \bar{u}_3 = -3.828 \bar{v}_3$$

$$(3) \quad P = \frac{EA}{l} \left[-\bar{u}_3 + \bar{u}_4 \right]$$

$$(4) \quad 0 = \frac{EA}{l} \left[\bar{v}_4 \right] \therefore \bar{v}_4 = 0$$

add (1) and (3) to eliminate \bar{u}_4 + subst (2) into it.

$$\frac{Pl}{EA} = \frac{5}{12} \bar{u}_3 + \frac{5}{12} \bar{v}_3 \rightarrow \bar{u}_3 + \bar{v}_3 = \frac{1}{.3536} \frac{Pl}{AE}$$

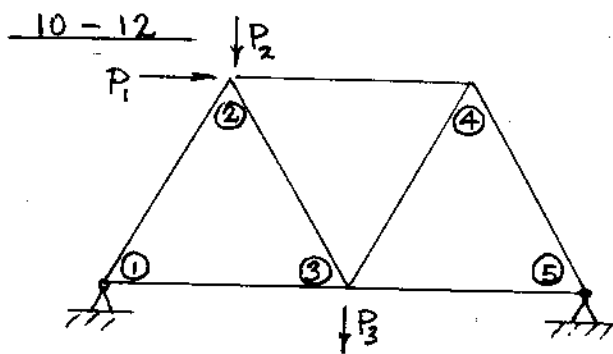
$$\bar{u}_3 - \frac{1}{3.828} \bar{u}_3 = \frac{Pl}{.3536 AE} \therefore \bar{u}_3 = 3.828 \frac{Pl}{AE}$$

$$\bar{v}_3 = -\frac{Pl}{AE}$$

$$\bar{u}_4 = 4.828 \frac{Pl}{AE}$$

Complete Equation

$$\begin{Bmatrix} \bar{F}_{1x} \\ \bar{F}_{1y} \\ \bar{F}_{2x} \\ \bar{F}_{2y} \\ \bar{F}_{3x} \\ \bar{F}_{3y} \\ \bar{F}_{4x} \\ \bar{F}_{4y} \end{Bmatrix} = \frac{EA}{l} \begin{bmatrix} (1+\frac{5}{12}) & \frac{5}{12} & -1 & 0 & -\frac{5}{12} & -\frac{5}{12} & 0 & 0 \\ \frac{5}{12} & (1+\frac{5}{12}) & 0 & 0 & -\frac{5}{12} & -\frac{5}{12} & 0 & -1 \\ -1 & 0 & 1 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 1 & 0 & -1 & 0 & 0 \\ -\frac{5}{12} & -\frac{5}{12} & 0 & 0 & (1+\frac{5}{12}) & \frac{5}{12} & -1 & 0 \\ -\frac{5}{12} & -\frac{5}{12} & 0 & -1 & \frac{5}{12} & (1+\frac{5}{12}) & 0 & 0 \\ 0 & 0 & 0 & 0 & -1 & 0 & 1 & 0 \\ 0 & -1 & 0 & 0 & 0 & 0 & 0 & 1 \end{bmatrix} \begin{Bmatrix} \bar{u}_1 = 0 \\ \bar{v}_1 = 0 \\ \bar{u}_2 = 0 \\ \bar{v}_2 = 0 \\ \bar{u}_3 \\ \bar{v}_3 \\ \bar{u}_4 \\ \bar{v}_4 \end{Bmatrix}$$



Elements 1-2, 3-4, $\alpha = 60^\circ$ $C = .5$, $S = .866$, $CS = .433$
 $C^2 = .250$ $S^2 = .750$

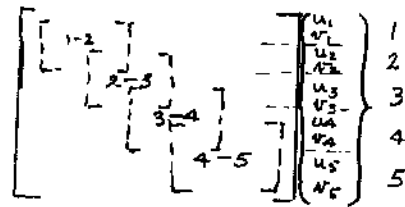
$$\bar{k} = \frac{EA}{l} \begin{bmatrix} .250 & .433 & -.250 & -.433 \\ .433 & .750 & -.433 & -.750 \\ -.250 & -.433 & .250 & .433 \\ .750 & .750 & .433 & .750 \end{bmatrix}$$

Elements 2-3, 4-5 $\alpha = 300^\circ$ $C = .5$ $S = -.866$ $CS = -.433$
 $C^2 = .250$ $S^2 = .750$

$$\bar{k} = \frac{EA}{l} \begin{bmatrix} .250 & -.433 & -.250 & .433 \\ .433 & .750 & .433 & -.750 \\ -.250 & .433 & .250 & -.433 \\ .750 & -.750 & .433 & .750 \end{bmatrix}$$

Element 1-3, 2-4, 3-5 $\alpha = 0$ $C = 1$ $S = 0$ $CS = 0$
 $C^2 = 1$ $S^2 = 0$

$$\bar{k} = \frac{EA}{l} \begin{bmatrix} 1 & 0 & -1 & 0 \\ 0 & 0 & 0 & 0 \\ -1 & 0 & 1 & 0 \\ 0 & 0 & 0 & 0 \end{bmatrix}$$



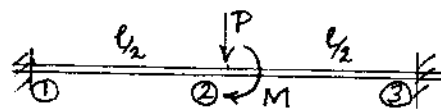
$$\begin{bmatrix} & & & & \\ & 10 & & -10 & \\ & 00 & & 00 & \\ & -10 & & 10 & \\ & 00 & & 00 & \end{bmatrix} \begin{matrix} 1 \\ 2 \\ 3 \\ 4 \\ 5 \end{matrix}$$

Element 2-4

$$\begin{bmatrix} & & & & \\ & & & & \\ & & 10 & & -10 \\ & & 00 & & 00 \\ & & -10 & & 10 \\ & & 00 & & 00 \end{bmatrix} \begin{matrix} 1 \\ 2 \\ 3 \\ 4 \\ 5 \end{matrix}$$

Element 3-5

10-13



From Example 10.5-1 and noting that the ends 1 and 3 are fixed, the stiffness matrix and displacement vectors are:

$$\begin{Bmatrix} F_2 \\ M_2 \\ F_3 \\ M_3 \end{Bmatrix} = \frac{8EI}{l^3} \begin{bmatrix} 24 & 0 & -12 & 3l \\ 0 & 2l^2 & -3l & .5l^2 \\ -12 & -3l & 12 & -3l \\ 3l & .5l^2 & -3l & l^2 \end{bmatrix} \begin{Bmatrix} \bar{u}_2 \\ \theta_2 \\ 0 \\ 0 \end{Bmatrix}$$

$$\begin{Bmatrix} F_2 \\ M_2 \end{Bmatrix} = \begin{Bmatrix} -P \\ -M \end{Bmatrix} = \frac{8EI}{l^3} \begin{bmatrix} 24 & 0 \\ 0 & 2l^2 \end{bmatrix} \begin{Bmatrix} \bar{u}_2 \\ \theta_2 \end{Bmatrix}$$

$$\therefore \bar{u}_2 = -\frac{Pl^3}{192EI} \quad \theta_2 = -\frac{Ml}{16EI}$$

$$M_3 = \frac{8EI}{l^3} [3l \bar{u}_2 + .5l^2 \theta_2] = \frac{8EI}{l^3} \left[3l \left(-\frac{Pl^3}{192EI} \right) + .5l^2 \left(-\frac{Ml}{16EI} \right) \right]$$

10-14

Refer to Prob 10.5-1 with $u_3 = \theta_3 = 0$

$$\left[-\frac{\omega^2 ml}{840} \begin{bmatrix} 312 & 0 \\ 0 & 2l^2 \end{bmatrix} + \frac{8EI}{l^3} \begin{bmatrix} 24 & 0 \\ 0 & 2l^2 \end{bmatrix} \right] \begin{Bmatrix} u_2 \\ \theta_2 \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \end{Bmatrix}$$

$$\text{Let } \lambda = \left(\frac{\omega^2 ml}{840} \right) \left(\frac{l^3}{8EI} \right) = \frac{ml^4 \omega^2}{6720EI}$$

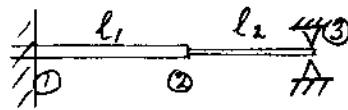
$$\begin{vmatrix} (24 - 312\lambda) & 0 \\ 0 & 2l^2(1 - \lambda) \end{vmatrix} = 0 \quad \begin{aligned} \lambda_1 &= \frac{24}{312} = \frac{ml^4 \omega_1^2}{6720EI} \\ \lambda_2 &= 1 = \frac{ml^4 \omega_2^2}{6720EI} \end{aligned}$$

$$\therefore \omega_1 = 22.74 \sqrt{\frac{EI}{ml^4}} \rightarrow 1.6\% \text{ high}$$

$$\omega_2 = 81.98 \sqrt{\frac{EI}{ml^4}} \rightarrow 33\% \text{ high.}$$

$$\text{Exact values } \begin{cases} \omega_1 = 22.37 \sqrt{\frac{EI}{ml^4}} \\ \omega_2 = 61.67 \sqrt{\frac{EI}{ml^4}} \end{cases}$$

10-15



Element 1-2

$$\bar{k} = \frac{EI_1}{l_1^3} \begin{bmatrix} R & 0 & 0 \\ 0 & 12 & -6l_1 \\ 0 & -6l_1 & 4l_1^2 \end{bmatrix} \begin{Bmatrix} \bar{u}_2 \\ \bar{v}_2 \\ \theta_2 \end{Bmatrix} \quad \bar{m} = \frac{m_1 l_1}{420} \begin{bmatrix} 156 & -22l_1 \\ -22l_1 & 4l_1^2 \end{bmatrix} \begin{Bmatrix} \ddot{v}_2 \\ \ddot{\theta}_2 \end{Bmatrix}$$

Element 2-3

$$\bar{k} = \frac{EI_2}{l_2^3} \begin{bmatrix} R & 0 & 0 & -R & 0 & 0 \\ 0 & 12 & 6l_2 & 0 & -12 & 6l_2 \\ 0 & 6l_2 & 4l_2^2 & 0 & -6l_2 & 2l_2^2 \\ -R & 0 & 0 & R & 0 & 0 \\ 0 & -12 & -6l_2 & 0 & 12 & -6l_2 \\ 0 & 6l_2 & 2l_2^2 & 0 & -6l_2 & 4l_2^2 \end{bmatrix} \begin{Bmatrix} \bar{u}_2=0 \\ \bar{v}_2 \\ \theta_2 \\ \bar{u}_3 \\ \bar{v}_3 \\ \theta_3 \end{Bmatrix}$$

$$\bar{m} = \frac{m_2 l_2}{420} \begin{bmatrix} N & 0 & 0 & \frac{1}{2}N & 0 & 0 \\ 0 & 156 & 22l_2 & 0 & 54 & -13l_2 \\ 0 & 22l_2 & 4l_2^2 & 0 & 13l_2 & -3l_2^2 \\ \frac{1}{2}N & 0 & 0 & N & 0 & 0 \\ 0 & 54 & 13l_2 & 0 & 156 & 22l_2 \\ 0 & -13l_2 & -3l_2^2 & 0 & -22l_2 & 4l_2^2 \end{bmatrix} \begin{Bmatrix} \ddot{u}_2=0 \\ \ddot{v}_2 \\ \ddot{\theta}_2 \\ \ddot{u}_3=0 \\ \ddot{v}_3=0 \\ \ddot{\theta}_3 \end{Bmatrix}$$

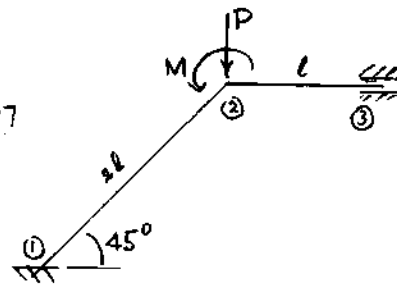
Free vibr. eq.

$$\frac{1}{420} \begin{bmatrix} 156(m_1 l_1 + m_2 l_2) & 22(m_2 l_2^2 - m_1 l_1^2) & -13 m_2 l_2^2 \\ 22(m_2 l_2^2 - m_1 l_1^2) & 4(m_2 l_2^3 + m_1 l_1^3) & -3 m_2 l_2^3 \\ -13 m_2 l_2^3 & -3 m_2 l_2^3 & 4 m_2 l_2^3 \end{bmatrix} \begin{Bmatrix} \ddot{v}_2 \\ \ddot{\theta}_2 \\ \ddot{\theta}_3 \end{Bmatrix}$$

$$+ E \begin{bmatrix} 12(\frac{I_2}{l_2^3} + \frac{I_1}{l_1^3}) & 6(\frac{I_2}{l_2^2} - \frac{I_1}{l_1^2}) & 6 \frac{I_2}{l_2^2} \\ 6(\frac{I_2}{l_2^2} - \frac{I_1}{l_1^2}) & 4(\frac{I_2}{l_2} + \frac{I_1}{l_1}) & 2 \frac{I_2}{l_2} \\ 6 \frac{I_2}{l_2^2} & 2 \frac{I_2}{l_2} & 4 \frac{I_2}{l_2} \end{bmatrix} \begin{Bmatrix} \bar{v}_2 \\ \theta_2 \\ \theta_3 \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \\ 0 \end{Bmatrix}$$

10-16

Element 1-2 $\alpha = 45^\circ$ $C = .707$ $S = .707$
 $C^2 = S^2 = CS = .50$
 $l_{12} = 2l$



$$\bar{k} = \left(\frac{EI}{8l^3} \right) \begin{bmatrix} (.5R+6) & .50(R-12) & 12l \cdot .707 \\ .50(R-12) & .5(R+12) & -12l \cdot .707 \\ 12l \cdot .707 & -12l \cdot .707 & 16l^2 \end{bmatrix} \begin{Bmatrix} \bar{u}_2 \\ \bar{v}_2 \\ \Theta_2 \end{Bmatrix} \quad \text{All } R_{1-2} = \frac{A(2l)^2}{I} = \frac{4Al^2}{I}$$

Element 2-3 $\alpha = 0$ $C = 1$, $S = 0$, $l_{23} = l$

$$\bar{k} = \left(\frac{EI}{l^3} \right) \begin{bmatrix} R_{23} & 0 & 0 \\ 0 & 12 & 6l \\ 0 & 6l & 4l^2 \end{bmatrix} \begin{Bmatrix} \bar{u}_2 \\ \bar{v}_2 \\ \Theta_2 \end{Bmatrix} \quad \text{All } R_{2-3} = \frac{Al^2}{I} = \frac{1}{4} R_{1-2}$$

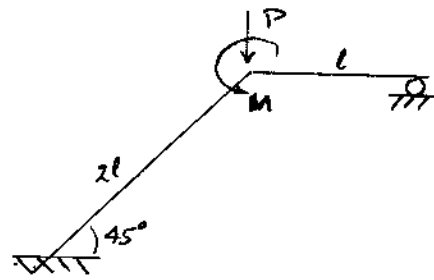
Assembled with all $R_s = R_{12} = R$

$$\bar{k} = \left(\frac{EI}{8l^3} \right) \begin{bmatrix} (.5R+2R+6) & .5(R-12) & .707 \times 12l \\ .5(R+12) & .5(R+12)+96 & -.707 \times 12l + 48l \\ 12 \times .707 l & -.707 \times 12l + 48l & 16l^2 + 32l^2 \end{bmatrix} \begin{Bmatrix} \bar{u}_2 \\ \bar{v}_2 \\ \Theta_2 \end{Bmatrix}$$

$$\begin{Bmatrix} 0 \\ -P \\ M \end{Bmatrix} = \left(\frac{EI}{8l^3} \right) \begin{bmatrix} (2.5R+6) & .5(R-12) & 8.484l \\ .5(R+12) & (.5R+102) & 39.52l \\ 8.484l & 39.52l & 48l^2 \end{bmatrix} \begin{Bmatrix} \bar{u}_2 \\ \bar{v}_2 \\ \Theta_2 \end{Bmatrix} \quad R = \frac{4Al^2}{I}$$

Element 1-2

$$\bar{k} = \left(\frac{EI}{8l^3} \right) \begin{bmatrix} .5R+6 & .5(R-12) & 8.484l \\ .5(R-12) & .5(R+12) & -8.484l \\ 8.484l & -8.484l & 16l^2 \end{bmatrix}$$



Element 2-3

$$\bar{k} = \left(\frac{EI}{l^3} \right) \begin{bmatrix} R_{2-3} & 0 & 0 \\ 0 & 3 & 3l \\ 0 & 3l & 3l^2 \end{bmatrix} = \left(\frac{EI}{8l^3} \right) \begin{bmatrix} 2R & 0 & 0 \\ 0 & 24 & 24l \\ 0 & 24l & 24l^2 \end{bmatrix}$$

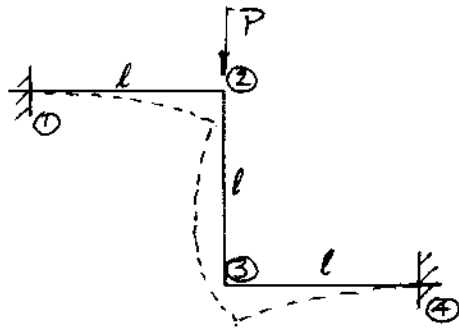
Assembled

$$\begin{Bmatrix} 0 \\ -P \\ M \end{Bmatrix} = \left(\frac{EI}{8l^3} \right) \begin{bmatrix} (2.5R+6) & .5(R-12) & 8.484l \\ .5(R-12) & (.5R+30) & 15.52l \\ 8.484l & 15.52l & 40l^2 \end{bmatrix} \begin{Bmatrix} \bar{u}_2 \\ \bar{v}_2 \\ \theta_2 \end{Bmatrix}$$

From 1st of above equation express \bar{u}_2 in terms of \bar{v}_2 and θ_2 and write

$$\begin{Bmatrix} -P \\ M \end{Bmatrix} = \left(\frac{EI}{8l^3} \right) \begin{bmatrix} k_{11} & k_{12} \\ k_{21} & k_{22} \end{bmatrix} \begin{Bmatrix} \bar{v}_2 \\ \theta_2 \end{Bmatrix}$$

10-18(a)



$$\begin{Bmatrix} F_{1x} \\ \vdots \\ \bar{M}_4 \end{Bmatrix} = \frac{EI}{l^3} \begin{bmatrix} 12 \times 12 \end{bmatrix} \begin{Bmatrix} \bar{u}_1 \\ \vdots \\ \theta_4 \end{Bmatrix}$$

Elements 1-2 and 3-4 $\alpha=0$ $C=1$, $S=0$

$$\bar{k}_{1-2} = \bar{k}_{3-4} = \frac{EI}{l^3} \begin{bmatrix} R & 0 & 0 & -R & 0 & 0 \\ 0 & 12 & 6l & 0 & -12 & 6l \\ 0 & 6l & 4l^2 & 0 & -6l & 2l^2 \\ -R & 0 & 0 & R & 0 & 0 \\ 0 & -12 & -6l & 0 & 12 & -6l \\ 0 & 6l & 2l^2 & 0 & -6l & 4l^2 \end{bmatrix} \quad \text{vectors} \left\{ \begin{array}{l} \bar{u}_1=0 \\ \bar{v}_1=0 \\ \theta_1=0 \\ \bar{u}_2 \\ \bar{v}_2 \\ \theta_2 \end{array} \right\} \text{ or } \left\{ \begin{array}{l} \bar{u}_3 \\ \bar{v}_3 \\ \theta_3 \\ \bar{u}_4=0 \\ \bar{v}_4=0 \\ \theta_4=0 \end{array} \right\}$$

Element 2-3 $\alpha=270^\circ$ or -90° $C=0$, $S=-1$

$$\bar{k}_{2-3} = \frac{EI}{l^3} \begin{bmatrix} 12 & 0 & 6l & -12 & 0 & 6l \\ 0 & R & 0 & 0 & -R & 0 \\ 6l & 0 & 4l^2 & -6l & 0 & 2l^2 \\ -12 & 0 & -6l & 12 & 0 & -6l \\ 0 & -R & 0 & 0 & R & 0 \\ 6l & 0 & 2l^2 & -6l & 0 & 4l^2 \end{bmatrix} \quad \text{vector} \left\{ \begin{array}{l} \bar{u}_2 \\ \bar{v}_2 \\ \theta_2 \\ \bar{u}_3 \\ \bar{v}_3 \\ \theta_3 \end{array} \right\}$$

Only displacements involving ② and ③ need to be considered. Thus we need only to add lower right quarter of \bar{k}_{1-2} to upper left quarter of \bar{k}_{2-3} , and upper left qtr. of \bar{k}_{3-4} to lower right qtr. of \bar{k}_{2-3}

$$\begin{Bmatrix} \bar{F}_{1x}=0 \\ -P \\ \bar{M}_2=0 \\ \vdots \\ \bar{F}_{3x}=0 \\ \bar{F}_{3y}=0 \\ \bar{M}_3=0 \end{Bmatrix} = \frac{EI}{l^3} \begin{bmatrix} (R+12) & 0 & 6l & -12 & 0 & 6l \\ 0 & (R+12) & -6l & 0 & -R & 0 \\ 6l & -6l & 8l^2 & -6l & 0 & 2l^2 \\ -12 & 0 & -6l & (R+12) & 0 & -6l \\ 0 & -R & 0 & 0 & (R+12) & 6l \\ 6l & 0 & 2l^2 & -6l & 6l & 8l^2 \end{bmatrix} \begin{Bmatrix} \bar{u}_2 \\ \bar{v}_2 \\ \theta_2 \\ \bar{u}_3 \\ \bar{v}_3 \\ \theta_3 \end{Bmatrix} \quad \text{symmetric matrix}$$

10-18(a) Cont.

For no axial extension $\bar{u}_2 = -\bar{u}_3 = 0$

this eliminates 1st and 4th columns

Then $F_{2x} = F_{3x} = 0$ and the 1st and 4th rows

requires that $\theta_3 = -\theta_2$ which eliminates
the 1st and 4th rows, leaving the 4x4
matrix

$$\begin{Bmatrix} -P \\ 0 \\ 0 \\ 0 \end{Bmatrix} = \frac{EI}{l^3} \begin{bmatrix} (R+12) & -6l & -R & 0 \\ -6l & 8l^2 & 0 & 2l^2 \\ -R & 0 & (R+12) & 6l \\ 0 & 2l^2 & 6l & 8l^2 \end{bmatrix} \begin{Bmatrix} \bar{u}_2 \\ \theta_2 \\ \bar{u}_3 \\ \theta_3 \end{Bmatrix} \quad \text{symmetric}$$

Since $\bar{u}_3 = \bar{u}_2$ add 3rd column to 1st column.

Also since $\theta_3 = -\theta_2$, subtract 4th col. from 2nd col.

This also reduces the matrix to a 2x2

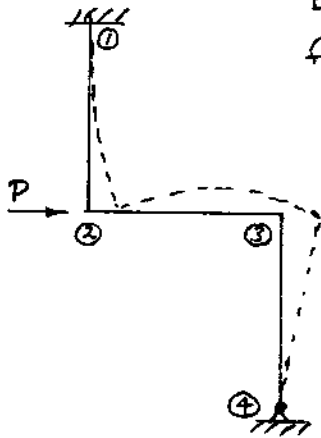
$$\begin{Bmatrix} -P \\ 0 \end{Bmatrix} = \frac{EI}{l^3} \begin{bmatrix} 12 & -6l \\ -6l & 6l^2 \end{bmatrix} \begin{Bmatrix} \bar{u}_2 \\ \theta_2 \end{Bmatrix}$$

Solving the two equations

$$l\theta_2 = \bar{u}_2$$

$$\boxed{\begin{aligned} \bar{u}_2 &= \frac{-Pl^3}{6EI} \\ \theta_2 &= \frac{-Pl^2}{6EI} \end{aligned}}$$

10-18 (b)



Elements 1-2 and 2-3 are already available from Prob 10-18(a) by interchanging 1-2 and 2-3

The new element 3-4 is obtained from Eq. (10-8-3) with $\alpha = -90^\circ$ $C=0$, $S=-1$

Element 3-4

$$\frac{EI}{l^3} \begin{bmatrix} 3 & 0 & 3l & -3 & 0 & 3l \\ 0 & R & 0 & 0 & -R & 0 \\ 3l & 0 & 3l^2 & 0 & 0 & 0 \\ -3 & 0 & -3l & 3 & 0 & -3l \\ 0 & -R & 0 & 0 & R & 0 \\ 0 & 0 & 0 & -3l & 0 & 3l^2 \end{bmatrix}$$

$$\begin{Bmatrix} \sum F_x = 0 \\ \sum F_y = 0 \\ \sum M_x = 0 \\ \sum F_x = 0 \\ \sum F_y = 0 \\ \sum M_x = 0 \\ \sum F_x = 0 \\ \sum F_y = 0 \\ \sum M_x = 0 \end{Bmatrix} = \frac{EI}{l^3} \begin{bmatrix} R-12 & 0 & 6l & 0 & 0 & 0 \\ 0 & -R & 0 & 0 & 0 & 0 \\ -6l & 0 & 2l^2 & 0 & 0 & 0 \\ (12+R) & 0 & -6l & -R & 0 & 0 \\ 0 & (12+R) & 6l & 0 & -12 & 6l \\ -6l & 6l & 8l^2 & 0 & -6l & 2l^2 \\ -R & 0 & 0 & (R+3) & 0 & 3l \\ 0 & -12 & -6l & 0 & (R+12) & -6l \\ 0 & 6l & 2l^2 & 3l & -6l & 7l^2 \\ 0 & 0 & 0 & -3 & 0 & -3l \\ 0 & -R & 0 & 0 & R & 0 \\ 0 & 0 & 0 & -3l & 0 & 3l^2 \end{bmatrix} \begin{Bmatrix} 0 \\ \bar{u}_2 \\ \bar{v}_2 \\ \theta_2 \\ \bar{u}_3 \\ \bar{v}_3 \\ \theta_3 \\ 0 \end{Bmatrix}$$

We need only the central part of the matrix

10-18 (b) Cont.

$$\bar{u}_2 = \bar{u}_3 = 0 \quad \therefore \text{eliminate cols 2 \& 5}$$

$$\begin{Bmatrix} \bar{F}_{2x} = P \\ \bar{M}_2 \\ \bar{F}_{3x} \\ \bar{M}_3 \end{Bmatrix} = \frac{EI}{l^3} \begin{bmatrix} (R+12) & -6l & -R & 0 \\ -6l & 8l^2 & 0 & 2l^2 \\ -R & 0 & (R+3) & 3l \\ 0 & 2l^2 & 3l & 7l^2 \end{bmatrix} \begin{Bmatrix} \bar{u}_2 \\ \theta_2 \\ \bar{u}_3 \\ \theta_3 \end{Bmatrix}$$

$$\bar{u}_2 = \bar{u}_3 \quad \therefore \text{add col 3 to 1}$$

$$\begin{Bmatrix} \frac{Pl^3}{EI} \\ 0 \\ 0 \\ 0 \end{Bmatrix} = \begin{bmatrix} 12 & -6l & 0 \\ -6l & 8l^2 & 2l^2 \\ 3 & 0 & 3l \\ 3l & 2l^2 & 7l^2 \end{bmatrix} \begin{Bmatrix} u_2 \\ \theta_2 \\ \theta_3 \end{Bmatrix} \quad \begin{array}{l} \frac{Pl^3}{EI} = 12\bar{u}_2 - 6l\theta_2 \quad \text{Eq. no. (1)} \\ -6l\bar{u}_2 + 8l^2\theta_2 + 2l^2\theta_3 = 0 \quad (2) \\ 3\bar{u}_2 + 3l\theta_3 = 0 \quad (3) \\ 3l\bar{u}_2 + 2l^2\theta_2 + 7l^2\theta_3 = 0 \quad (4) \end{array}$$

$$2 \times (4) + (2) = 0u_2 + 12l^2\theta_2 + 16l^2\theta_3 = 0 \quad \therefore \theta_2 = -\frac{4}{3}\theta_3$$

$$(3) \text{ gives } \bar{u}_2 = -l\theta_3 \quad \therefore = \frac{3}{4}l\theta_2$$

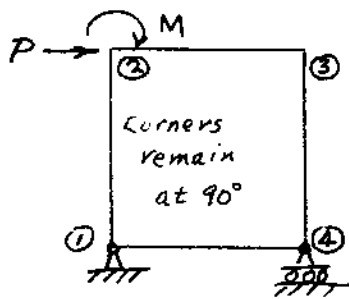
$$\text{Subst } \theta_2 = -\frac{4}{3}\theta_3 \text{ into (1)}$$

$$\frac{Pl^3}{EI} = 12u_2 - 6l\left(-\frac{4}{3}\theta_3\right) = 12\bar{u}_2 + 8l\theta_3 = 12\bar{u}_2 + 8\bar{u}_2 = 4\bar{u}_2$$

$$\therefore \bar{u}_2 = \frac{Pl^3}{4EI}, \quad \theta_3 = -\frac{Pl^2}{4EI}, \quad \theta_2 = \frac{Pl^2}{3EI}$$

Comparing with Prob(10-18(a)) the deflection under P is larger with pinned end, and the two rotations are dissimilar, and also larger. Conclusion: Pinned end leads to more flexible structure.

10-18 (c)



Element

1-2	$C = 0, S = 1$
2-3	$C = 1, S = 0$
3-4	$C = 0, S = -1$
4-1	$C = -1, S = 0$

Element stiffness

$$\textcircled{1}-\textcircled{2} \quad \begin{bmatrix} 12 & 0 & -6l & -12 & 0 & -6l \\ 0 & R & 0 & 0 & -R & 0 \\ -6l & 0 & 4l^2 & 6l & 0 & 2l^2 \\ -12 & 0 & 6l & 12 & 0 & 6l \\ 0 & -R & 0 & 0 & R & 0 \\ -6l & 0 & 2l^2 & 6l & 0 & 4l^2 \end{bmatrix}$$

$$\textcircled{2}-\textcircled{3} \quad \begin{bmatrix} R & 0 & 0 & -R & 0 & 0 \\ 0 & 12 & 6l & 0 & -12 & 6l \\ 0 & 6l & 4l^2 & 0 & -6l & 2l^2 \\ -R & 0 & 0 & R & 0 & 0 \\ 0 & -12 & -6l & 0 & 12 & -6l \\ 0 & 6l & 2l^2 & 0 & -6l & 4l^2 \end{bmatrix} = \textcircled{1}-\textcircled{4}$$

$$\textcircled{3}-\textcircled{4} \quad \begin{bmatrix} 12 & 0 & 6l & -12 & 0 & 6l \\ 0 & R & 0 & 0 & -R & 0 \\ 6l & 0 & 4l^2 & -6l & 0 & 2l^2 \\ -12 & 0 & -6l & 12 & 0 & -6l \\ 0 & -R & 0 & 0 & R & 0 \\ 6l & 0 & 2l^2 & -6l & 0 & 4l^2 \end{bmatrix}$$

$$\textcircled{4}-\textcircled{1} \quad \begin{bmatrix} R & 0 & 0 & -R & 0 & 0 \\ 0 & 12 & -6l & 0 & -12 & -6l \\ 0 & -6l & 4l^2 & 0 & 6l & 2l^2 \\ -R & 0 & 0 & R & 0 & 0 \\ 0 & -12 & 6l & 0 & 12 & 6l \\ 0 & -6l & 2l^2 & 0 & 6l & 4l^2 \end{bmatrix}$$

Displacements.

$$\bar{u}_1 = \bar{v}_1 = 0$$

$$\bar{v}_2 = \bar{v}_3 = 0$$

$$\bar{u}_2 = \bar{u}_3$$

$$\bar{v}_4 = 0$$

$$\bar{u}_4 = 0$$

Coordinates not zero

$$\bar{u}_2, \bar{u}_3, \theta_1, \theta_2, \theta_3, \theta_4$$

none of the θ_s need to

be equal, however

we can assume that $\bar{u}_2 = \bar{u}_3$

This leaves 5 unknowns.

10-18(c) Cont.

Assembled matrix (symmetric)

$$\begin{array}{c}
 \textcircled{1} \\
 \textcircled{2} \\
 \textcircled{3} \\
 \textcircled{4}
 \end{array}
 \left[\begin{array}{cccccc|cccc|c|c}
 (12+R) & 0 & -6l & -12 & 6 & -6l & -R & 0 & 0 & 0 & \bar{u}_1 = 0 \\
 0 & (12+R) & 6l & 0 & -R & 0 & 0 & -12 & 6l & 0 & \bar{u}_1 = 0 \\
 -6l & 6l & 8l^2 & 6l & 0 & 2l^2 & 0 & -6l & 2l^2 & \theta_1 & \theta_1 \\
 \hline
 -12 & 0 & 6l & (12+R) & 0 & 6l & -R & 0 & 0 & u_2 & \bar{u}_2 \\
 0 & -R & 0 & 0 & (12+R) & 6l & 0 & -12 & 6l & 0 & \bar{u}_2 = 0 \\
 -6l & 0 & 2l^2 & 6l & 6l & 8l^2 & 0 & -6l & 2l^2 & \theta_2 & \theta_2 \\
 \hline
 & & & & & & -R & 0 & 0 & (12+R) & 0 & 6l & -12 & 0 & 6l & \bar{u}_3 = \bar{u}_2 \\
 & & & & & & 0 & -12 & -6l & 0 & (12+R) & -6l & 0 & -R & 0 & 0 \\
 & & & & & & 0 & 6l & 2l^2 & 6l & -6l & 8l^2 & -6l & 0 & 2l^2 & \theta_3 \\
 \hline
 -R & 0 & 0 & & & & -12 & 0 & -6l & (12+R) & 0 & -6l & 0 & 0 & 0 & \bar{u}_4 = 0 \\
 0 & -12 & -6l & & & & 0 & -R & 0 & 0 & (12+R) & -6l & 0 & 0 & 0 & \bar{u}_4 = 0 \\
 0 & 6l & 2l^2 & & & & 6l & 0 & 2l^2 & -6l & -6l & 8l^2 & 0 & 0 & 0 & \theta_4
 \end{array} \right]$$

$\bar{u}_3 = \bar{u}_2$ ∴ add col. 7 to 4 and rewrite

$$\left\{ \begin{array}{c} \bar{F}_{1x} \\ \bar{F}_{1y} \\ 0 \\ P \\ 0 \\ -M \\ 0 \\ 0 \\ 0 \\ \bar{F}_{4x} \\ \bar{F}_{4y} \\ 0 \end{array} \right\} = \frac{EI}{l^3} \left[\begin{array}{ccccc} -6l & -12 & -6l & 0 & 0 \\ 6l & 0 & 0 & 0 & 6l \\ 8l^2 & 6l & 2l^2 & 0 & 2l^2 \\ 6l & 12 & 6l & 0 & 0 \\ 0 & 0 & 6l & 6l & 0 \\ 2l^2 & 6l & 8l^2 & 2l^2 & 0 \\ 0 & 12 & 0 & 6l & 6l \\ 0 & 0 & -6l & -6l & 0 \\ 0 & 6l & 2l^2 & 8l^2 & 2l^2 \\ 0 & -12 & 0 & -6l & -6l \\ -6l & 0 & 0 & 0 & -6l \\ 2l^2 & 6l & 0 & 2l^2 & 8l^2 \end{array} \right] \left\{ \begin{array}{c} \theta_1 \\ \bar{u}_2 \\ \theta_2 \\ \theta_3 \\ \theta_4 \end{array} \right\}$$

12x5 matrix

5th and 8th equations are identical — both give $\theta_2 = -\theta_3$
 (remove 5th & 8th eqs. after subtracting
 4th col from 3rd col.)

10-18(c) Cont

Rewrite eq with col 4 subtracted from col 3 and 5th & 8th eqs. removed

$$\begin{Bmatrix} \bar{F}_{1x} \\ \bar{F}_{1y} \\ 0 \\ P \\ -M \\ 0 \\ 0 \\ \bar{F}_{4x} \\ \bar{F}_{4y} \\ 0 \end{Bmatrix} = \frac{EI}{l^3} \begin{bmatrix} -6l & -12 & -6l & 0 \\ 6l & 0 & 0 & 6l \\ 8l^2 & 6l & 2l^2 & 2l^2 \\ 6l & 12 & 6l & 0 \\ 2l^2 & 6l & 6l^2 & 0 \\ 0 & 12 & -6l & 6l \\ 0 & 6l & -6l^2 & 2l^2 \\ 0 & -12 & 6l & -6l \\ -6l & 0 & 0 & -6l \\ 2l^2 & 6l & -2l^2 & 8l^2 \end{bmatrix} \begin{Bmatrix} \theta_1 \\ \bar{u}_2 \\ \theta_2 \\ \theta_4 \end{Bmatrix} \quad 10 \times 4 \text{ matrix}$$

Eqs. for \bar{F}_{1y} and \bar{F}_{4y} are equal but of opposite signs

$$\therefore \bar{F}_{1y} = -\bar{F}_{4y} \quad (\text{remove these two eqs.})$$

6th eq and \bar{F}_{4x} are of opposite signs but 6th eq = 0

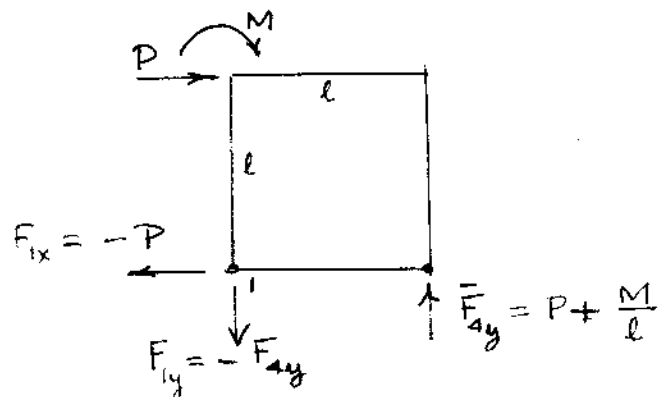
$$\therefore \bar{F}_{4x} = 0 \quad (\text{remove these two eqs.})$$

Eqs for \bar{F}_{1x} and P are of opposite signs

$$\therefore F_{1x} = -P$$

Moment about ① gives

$$F_{4y} l = Pl + M$$



The above is just for checking, since these results can be obtained from statics.

10-18(c) Cont.

Rewrite unused eqs from 10×4 matrix

$$\frac{Pl^3}{EI} = 6l\theta_1 + 12\bar{u}_2 + 6l\theta_2 \quad (1)$$

$$-\frac{Ml^2}{EI} = 2l\theta_1 + 6\bar{u}_2 + 6l\theta_2 \quad (2)$$

$$0 = 8l^2\theta_1 + 6l\bar{u}_2 + 2l^2\theta_2 + 2l^2\theta_4 \quad (3)$$

$$0 = 2l^2\theta_1 + 6l\bar{u}_2 - 2l^2\theta_2 + 8l^2\theta_4 \quad (4)$$

Eliminate θ_4 betw. (3) & (4)

$$0 = 30l^2\theta_1 + 18l\bar{u}_2 + 10l^2\theta_2 \quad (5)$$

Eqs. (1) (2) & (5) has 3 unknowns with 3 eqs.

In (5) solve for $l\theta_1$ in terms of \bar{u}_2 and θ_2

$$l\theta_1 = -(.60\bar{u}_2 + .3333l\theta_2) \text{ subst. into (1) and (2)}$$

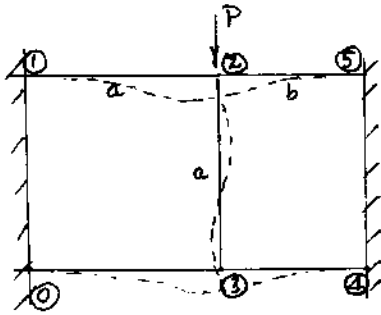
$$\frac{Pl^3}{EI} = (12 - 3.60)\bar{u}_2 + (6l - 2l)\theta_2 = 8.40\bar{u}_2 + 4l\theta_2$$

$$-\frac{Ml^2}{EI} = (6 - 1.20)\bar{u}_2 + (6l - .666l)\theta_2 = 4.80\bar{u}_2 + 5.333l\theta_2$$

Eliminate θ_2 for eq. for \bar{u}_2 , then eliminate \bar{u}_2

$$\boxed{\begin{aligned}\bar{u}_2 &= \frac{1}{4.80} \left(\frac{Pl^3}{EI} + .75 \frac{Ml^2}{EI} \right) \\ \theta_2 &= \frac{-1}{5.333l} \left(\frac{Pl^3}{EI} + 1.75 \frac{Ml^2}{EI} \right)\end{aligned}}$$

10-18(d)



$$\text{let } a = l_1 = l$$

$$b = l_2 = \frac{b}{a} l_1$$

$$\bar{u}_2 = \bar{u}_3 = 0$$

$$\bar{v}_2 = \bar{v}_3$$

$$\left\{ \begin{array}{l} \bar{u}_1 = 0 \\ \bar{v}_1 = 0 \\ \theta_1 = 0 \\ \bar{u}_2 \\ \bar{v}_2 \\ \theta_2 \\ \bar{u}_3 \\ \bar{v}_3 \\ \theta_3 \\ \bar{u}_4 = 0 \\ \bar{v}_4 = 0 \\ \theta_4 = 0 \end{array} \right.$$

$$\bar{k}_{1-2} = \frac{EI}{l_1^3} \left[\begin{array}{ccc|ccc} -R & 0 & 0 & 0 & -12 & 6l \\ 0 & -12 & 6l & 0 & 6l & -12 \\ 0 & 6l & -12 & 0 & 0 & 0 \\ \hline R & 0 & 0 & 0 & 12 & -6l \\ 0 & 12 & -6l & 0 & -6l & 12 \\ 0 & -6l & 12 & 0 & 4l^2 & -6l \end{array} \right] = \bar{k}_{3-2}$$

$$\bar{k}_{2-3} = \frac{EI}{l_1^3} \left[\begin{array}{ccc|ccc} 12 & 0 & 6l & -12 & 0 & 6l \\ 0 & R & 0 & 0 & -R & 0 \\ 6l & 0 & 4l^2 & -6l & 0 & 2l^2 \\ \hline -12 & 0 & -6l & 12 & 0 & -6l \\ 0 & -R & 0 & 0 & R & 0 \\ 6l & 0 & 2l^2 & -6l & 0 & 4l^2 \end{array} \right]$$

$$\bar{k}_{3-4} = \frac{EI}{l_2^3} \left[\begin{array}{ccc|ccc} R & 0 & 0 & 0 & 0 & 0 \\ 0 & 12 & 6l_2 & 0 & -12 & 6l_2 \\ 0 & 6l_2 & 4l_2^2 & 0 & -6l_2 & 2l_2^2 \\ \hline -R & 0 & 0 & 0 & 0 & 0 \\ 0 & -12 & -6l_2 & 0 & 12 & -6l_2 \\ 0 & 6l_2 & 2l_2^2 & 0 & -6l_2 & 4l_2^2 \end{array} \right] = \frac{EI}{l^3} \left[\begin{array}{ccc|ccc} R(\frac{a}{b})^3 & 0 & 0 & 0 & 0 & 0 \\ 0 & 12(\frac{a}{b})^3 & 6l(\frac{a}{b})^2 & 0 & -12(\frac{a}{b})^3 & -6l(\frac{a}{b})^2 \\ 0 & 6l(\frac{a}{b})^2 & 4l^2(\frac{a}{b}) & 0 & -6l(\frac{a}{b})^2 & 2l^2(\frac{a}{b}) \\ \hline -R(\frac{a}{b})^3 & 0 & 0 & 0 & 0 & 0 \\ 0 & -12(\frac{a}{b})^3 & -6l(\frac{a}{b})^2 & 0 & 12(\frac{a}{b})^3 & 6l(\frac{a}{b})^2 \\ 0 & 6l(\frac{a}{b})^2 & 2l^2(\frac{a}{b}) & 0 & -6l(\frac{a}{b})^2 & 4l^2(\frac{a}{b}) \end{array} \right] = \bar{k}_{2-5}$$

$$= \bar{k}_{3-4}$$

For the assembly we can first consider a matrix space of 6×6 each of which is composed of a 3×3 elements. i.e. \bar{k}_{2-3} has 4 sub-matrices, each corner of which has a 3×3 matrix. Since ①, ④, ⑤ all have zero deflections, we will be concerned only with the displacements of ② and ③ which occupies the central portion of the 6×6 space.

10-18(d) Cont.

0	1	2	3	4	5
0,0			0,3		
		\bar{k}_{12}			
		2,2	\bar{k}_{23}	2,5	
3,0			3,3		
			\bar{k}_{34}		
		5,2		5,5	

Sub matrices of \bar{k}_{2-5}

0	2,2	2,5
1	5,2	5,5
2	shown in 6x6 space	

3	←	
4	sub matrices of \bar{k}_{0-3}	
5	0,0	0,3
	3,0	3,3

← shown in 6x6 space

The matrix of interest which is indicated by the square in double lines also involves the lower right quarter of \bar{k}_{12} , all of \bar{k}_{23} and upper left quarter of \bar{k}_{3-4}

$$\frac{EI}{l^3} \begin{bmatrix} R+12 & 0+0 & 0+6l & -12 & 0 & 6l \\ +R(\frac{a}{b})^3 & 0 & 0 & & & \\ 0+0 & 12+R & -6l+0 & 0 & -R & 0 \\ 0 & 12(\frac{a}{b})^3 & 6l(\frac{a}{b})^2 & & & \\ 0+6l & -6l+0 & 4l^2+4l^2 & -6l & 0 & 2l^2 \\ 0 & 6l(\frac{a}{b})^2 & 4l^2(\frac{a}{b}) & & & \\ -12 & 0 & -6l & R+12 & 0+0 & 0-6l \\ & & & R(\frac{a}{b})^3 & 0 & 0 \\ 0 & -R & 0 & 0+0 & 12+R & -6l+0 \\ & & & 0 & 12(\frac{a}{b})^3 & 6l(\frac{a}{b})^2 \\ 6l & 0 & 2l^2 & 0-6l & -6l+0 & 4l^2+4l^2 \\ & & & & 6l(\frac{a}{b})^2 & 4l^2(\frac{a}{b}) \end{bmatrix} \begin{Bmatrix} \bar{u}_1 \\ \bar{u}_2 \\ \theta_2 \\ \bar{u}_3 \\ \bar{u}_3 \\ \theta_3 \end{Bmatrix}$$

10-18 (d) Cont

Note that
$$\begin{cases} \bar{u}_2 = \bar{u}_3 = 0 \\ \bar{v}_2 = \bar{v}_3 \\ \theta_2 = \theta_3 \end{cases}$$

cross out cols. 1 and 4

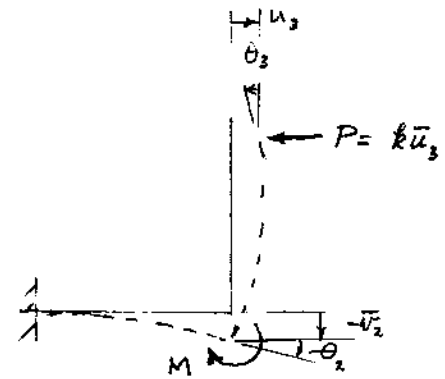
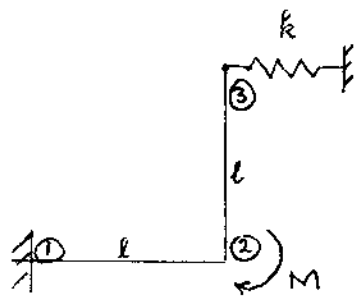
add col. 5 to 2

add col. 6 to 3

Resulting equations

$$\begin{Bmatrix} \bar{F}_{2y} = -P \\ \bar{M}_2 = 0 \end{Bmatrix} = \frac{EI}{\alpha^3} \begin{bmatrix} (12)\left[1 + \left(\frac{a}{b}\right)^3\right] & 6l\left[\left(\frac{a}{b}\right)^2 - 1\right] \\ 6l\left[\left(\frac{a}{b}\right)^2 - 1\right] & 4l^2\left[2.5 + \left(\frac{a}{b}\right)\right] \end{bmatrix} \begin{Bmatrix} \bar{v}_2 \\ \theta_2 \end{Bmatrix}$$

10-18(c)



$$\begin{Bmatrix} \bar{F}_{1x} \\ \bar{F}_{1y} \\ \bar{M}_1 \\ \bar{F}_{2x} \\ \bar{F}_{2y} \\ \bar{M}_2 \\ \bar{F}_{3x} \\ \bar{F}_{3y} \\ \bar{M}_3 \end{Bmatrix} = \frac{EI}{l^3} \begin{bmatrix} R & 0 & 0 & -R & 0 & 0 \\ 0 & 12 & 6l & 0 & -12 & 6l \\ 0 & 6l & 4l^2 & 0 & -6l & 2l^2 \\ \hline -R & 0 & 0 & R+12 & 0+0 & 0-6l \\ 0 & -12 & -6l & 0+0 & 12+R & -6l+0 \\ 0 & 6l & 2l^2 & 0+l & -6l+0 & 4l^2+4l^2 \\ \hline -12 & 0 & 6l & (12+\frac{kl^3}{EI}) & 0 & 6l \\ 0 & -R & 0 & 0 & R & 0 \\ -6l & 0 & 2l^2 & 6l & 0 & 4l^2 \end{bmatrix} \begin{Bmatrix} \bar{u}_1=0 \\ \bar{v}_1=0 \\ \theta_1=0 \\ \bar{u}_2=0 \\ \bar{v}_2 \\ \theta_2 \\ \bar{u}_3 \\ \bar{v}_3 \\ \theta_3 \end{Bmatrix}$$

$\bar{v}_3 = \bar{v}_2 \therefore$ add col. 8 to col. 5

$$\begin{Bmatrix} 0 \\ -M \\ 0 \\ 0 \end{Bmatrix} = \frac{EI}{l^3} \begin{bmatrix} 12 & -6l & 0 & 0 \\ -6l & 8l^2 & 6l & 2l^2 \\ 0 & 6l & (12+\frac{kl^3}{EI}) & 6l \\ 0 & 2l^2 & 6l & 4l^2 \end{bmatrix} \begin{Bmatrix} \bar{v}_2 \\ \theta_2 \\ \bar{u}_3 \\ \theta_3 \end{Bmatrix} \quad \text{Sym. } 4 \times 4$$

4 eqs, 4 unknowns

10-18(e) Cont.

$$\textcircled{1} \quad 0 = 12 \bar{v}_2 - 6l\theta_2$$

$$\textcircled{2} \quad -\frac{Ml^3}{EI} = -6l\bar{v}_2 + 8l^2\theta_2 + 6l\bar{u}_3 + 2l^2\theta_3$$

$$\textcircled{3} \quad 0 = 6l\theta_2 + \left(12 + \frac{kl^3}{EI}\right)\bar{u}_3 + 6l\theta_3$$

$$\textcircled{4} \quad 0 = 2l^2\theta_2 + 6l\bar{u}_3 + 4l^2\theta_3$$

From $\textcircled{1}$ $\bar{v}_2 = \frac{l}{2}\theta_2$ sub into $\textcircled{2}$

$$\textcircled{2'} \quad -\frac{Ml^3}{EI} = 5l^2\theta_2 + 6l\bar{u}_3 + 2l^2\theta_3$$

From $\textcircled{4}$ $\theta_2 = -\frac{3}{l}\bar{u}_3 - 2\theta_3$

Sub. into $\textcircled{3}$ $0 = -6l\left(\frac{3}{2}\bar{u}_3 + 2\theta_3\right) + \left(12 + \frac{kl^3}{EI}\right)\bar{u}_3 + 6l\theta_3$
 $= (-18 + 12 + \frac{kl^3}{EI})\bar{u}_3 - 12l\theta_3 + 6l\theta_3$

$$\therefore 6l\theta_3 = \left(-6 + \frac{kl^3}{EI}\right)\bar{u}_3$$

$$-\frac{Ml^3}{EI} = 5l^2\left(-\frac{3}{l}\bar{u}_3 - 2\theta_3\right) + 6l\bar{u}_3 + 2l^2\theta_3 = -9l\bar{u}_3 - 8l^2\theta_3$$

Sub for θ_3 in above eq.

$$-\frac{Ml^3}{EI} = -9l\bar{u}_3 - 8l^2\left(\frac{-6 + \frac{kl^3}{EI}}{6l}\right)\bar{u}_3 = \left(-l - \frac{4}{3}\frac{kl^3}{EI}\right)\bar{u}_3$$

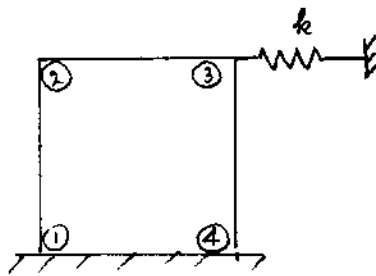
$$\therefore \bar{u}_3 = \frac{Ml^2}{\left(1 + \frac{4}{3}\frac{kl^3}{EI}\right)EI}$$

$$\theta_3 = \frac{1}{6l} \frac{\left(-6 + \frac{kl^3}{EI}\right)}{\left(1 + \frac{4}{3}\frac{kl^3}{EI}\right)} \frac{Ml^2}{EI}$$

$$\theta_2 = -\frac{3}{l}\bar{u}_3 - 2\theta_3$$

$$\bar{v}_2 = -\frac{3}{2}\bar{u}_3 - l\theta_2$$

10-19(a)



Stiffness matrix

$$\frac{EI}{l^3} \begin{bmatrix} (12+R) & 0 & 6l & -R & 0 & 0 \\ 0 & (12+R) & 6l & 0 & -12 & 6l \\ 6l & 6l & 8l^2 & 0 & -6l & 2l^2 \\ -R & 0 & 0 & (12+R+\frac{Rl^2}{EI}) & 0 & 6l \\ 0 & -12 & -6l & 0 & (12+R) & -6l \\ 0 & 6l & 2l^2 & 6l & -6l & 8l^2 \end{bmatrix} \begin{Bmatrix} \bar{u}_2 \\ \bar{v}_2 = 0 \\ \theta_2 \\ \bar{u}_3 = \bar{u}_2 \\ \bar{v}_3 = 0 \\ \theta_3 \end{Bmatrix}$$

Upper left qtr = lower right qtr of elem. ①-② plus upper left qtr of elem. ②-③.

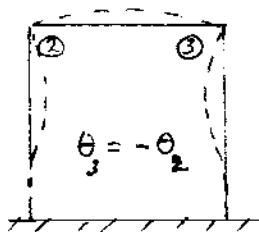
Lower right qtr = upper left qtr of elem. ③-④ plus lower right qtr of elem. ②-③.

mass matrix $\frac{ml}{420}$

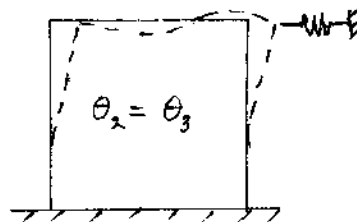
$$\begin{bmatrix} (156+N) & 0 & 22l & \frac{1}{2}N & 0 & 0 \\ 0 & (156+N) & 22l & 0 & -54 & -13l \\ 22l & 22l & 8l^2 & 0 & 13l & -3l^2 \\ \frac{1}{2}N & 0 & 0 & (156+N) & 0 & 22l \\ 0 & -54 & -13l & 0 & (156+N) & -22l \\ 0 & -13l & -3l^2 & 22l & -22l & 8l^2 \end{bmatrix} \begin{Bmatrix} \ddot{u}_2 \\ \ddot{v}_2 \\ \ddot{\theta}_2 \\ \ddot{u}_3 \\ \ddot{v}_3 \\ \ddot{\theta}_3 \end{Bmatrix}$$

Assumptions

$\bar{u}_2 = \bar{u}_3$, $\bar{v}_2 = \bar{v}_3 = 0$, two modes $\begin{cases} \theta_3 = -\theta_2 \rightarrow \text{spring not involved} \\ \theta_3 = \theta_2 \end{cases}$



Spring not in action



10-19(a) Cont

General eq. of motion

$$\frac{ml}{420} \begin{bmatrix} (156+N) & 22l & \frac{1}{2}N & 0 \\ 22l & 8l^2 & 0 & -3l^2 \\ \frac{1}{2}N & 0 & (156+N) & 22l \\ 0 & -3l^2 & 22l & 8l^2 \end{bmatrix} \begin{Bmatrix} \ddot{u}_2 \\ \ddot{\theta}_2 \\ \ddot{u}_3 \\ \ddot{\theta}_3 \end{Bmatrix} + \frac{EI}{l^3} \begin{bmatrix} (12+R) & 6l & -R & 0 \\ 6l & 8l^2 & 0 & 2l^2 \\ -R & 0 & (12+R+\frac{kl^3}{EI}) & 6l \\ 0 & 2l^2 & 6l & 8l^2 \end{bmatrix} \begin{Bmatrix} \bar{u}_2 \\ \theta_2 \\ \bar{u}_3 \\ \theta_3 \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \\ 0 \\ 0 \end{Bmatrix}$$

Symmetric mode $\bar{u}_2 = \bar{u}_3 = 0$ \therefore cross out col. ①, col. ③
 $\theta_3 = -\theta_2$ row ①, row ③

$$\frac{ml}{420} \begin{bmatrix} 8l^2 & -3l^2 \\ -3l^2 & 8l^2 \end{bmatrix} \begin{Bmatrix} \ddot{\theta}_2 \\ \ddot{\theta}_3 \end{Bmatrix} + \frac{EI}{l^3} \begin{bmatrix} 8l^2 & 2l^2 \\ 2l^2 & 8l^2 \end{bmatrix} \begin{Bmatrix} \theta_2 \\ \theta_3 \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \end{Bmatrix}$$

$$\left[-\omega^2 \frac{ml}{420} (11) + \frac{EI}{l^3} (6) \right] l^2 \theta_2 = 0$$

$$\omega^2 = \frac{6 \times 420}{11} \frac{EI}{ml^4} \quad \omega = 15.14 \sqrt{\frac{EI}{ml^4}}$$

Antisym. mode $\bar{u}_2 = \bar{u}_3 \neq 0$, $\theta_2 = \theta_3$

$$\left[-\frac{\omega^2 ml}{420} \begin{bmatrix} (156+\frac{3}{2}N) & 22l \\ 22l & 5l^2 \end{bmatrix} + \frac{EI}{l^3} \begin{bmatrix} (12+\frac{kl^3}{EI}) & 6l \\ 6l & 10l^2 \end{bmatrix} \right] \begin{Bmatrix} \bar{u} \\ \theta \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \end{Bmatrix}$$

with $\lambda = \left(\frac{\omega^2 ml^4}{420 EI} \right)$ $N=140$ Characteristic eq. =

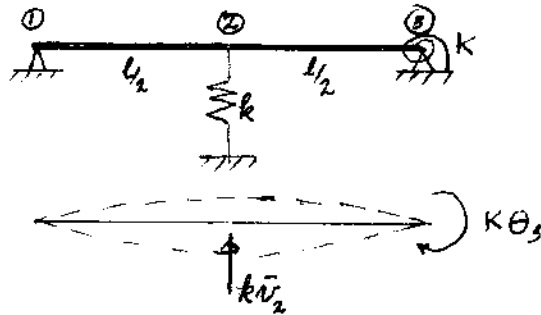
$$\begin{vmatrix} (12 + \frac{kl^3}{EI} - 366\lambda) & (6 - 22\lambda)l \\ (6 - 22\lambda)l & (10 - 5\lambda)l^2 \end{vmatrix} = 0$$

$$\lambda^2 - (2.6657 + .003715 \frac{kl^3}{EI}) \lambda + .062407 = 0$$

\therefore must decide on numerical value of $\frac{kl^3}{EI}$ before solving

If $\frac{kl^3}{EI} = 0$, then $\lambda_1 = .0236$ $\omega_1 = 3.149 \sqrt{EI/ml^4}$
 $\lambda_2 = 2.642$ $\omega_2 = 33.311 \sqrt{EI/ml^4}$ (Note numerical error on p279 text)

10-19(b)



$$\bar{u}_1 = \bar{u}_2 = \bar{u}_3 = 0$$

Element Stiffness 1-2 and 2-3 Eq. (10.2-1) with $l = \frac{l}{2}$

$$\frac{8EI}{l^3} \begin{bmatrix} 12 & 3l & -12 & 3l \\ 3l & l^2 & -3l & \frac{1}{2}l^2 \\ -12 & -3l & 12 + \frac{kl^3}{8EI} & -3l \\ 3l & \frac{1}{2}l^2 & -3l & (l^2 + \frac{kl^3}{8EI}) \end{bmatrix} \text{ displ.} = \begin{Bmatrix} \bar{v}_1 \\ \theta_1 \\ \bar{v}_2 \\ \theta_3 \end{Bmatrix} \text{ or } \begin{Bmatrix} \bar{v}_2 \\ \theta_2 \\ \bar{v}_3 \\ \theta_3 \end{Bmatrix}$$

The term $\frac{kl^3}{8EI}$ is used only in one of the \bar{v}_2 elements

Element Mass 1-2 and 2-3 Eq. (10.2-10) with $l = \frac{l}{2}$

$$\frac{ml}{840} \begin{bmatrix} 156 & 11l & 54 & -6.5l \\ 11l & l^2 & 6.5l & -7.5l^2 \\ 54 & 6.5l & 156 & -11l \\ -6.5l & -7.5l^2 & -11l & l^2 \end{bmatrix}$$

Assembled

$$\frac{8EI}{l^3} \begin{bmatrix} 12 & 3l & -12 & 3l \\ 3l & l^2 & -3l & .5l^2 \\ -12 & -3l & 12 + \frac{kl^3}{8EI} & 0 \\ 3l & .5l^2 & 0 & 2l^2 \end{bmatrix} \text{ displ.} = \begin{Bmatrix} \bar{v}_1 = 0 \\ \theta_1 \\ \bar{v}_2 \\ \theta_2 \\ \bar{v}_3 = 0 \\ \theta_3 \end{Bmatrix}$$

∴ delete cols 1 and 5 and rows 1 and 5

10-19(b) Cont.

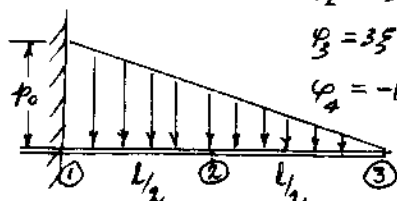
Assembled.

$$\frac{ml}{840} \begin{bmatrix} 156 & 11l & 54 & -6.5l \\ 11l & l^2 & 6.5l & -7.5l^2 \\ 54 & 6.5l & 312 & 0 \\ -6.5l & -7.5l^2 & 0 & 2l^2 \\ 54 & 6.5l & 156 & -11l \\ -6.5l & -7.5l^2 & -11l & l^2 \end{bmatrix}$$

Eq. of motion is a 4×4

$$\left[-\frac{\omega^2 ml}{840} \begin{bmatrix} l^2 & 6.5l & -7.5l^2 & 0 \\ 6.5l & 312 & 0 & -6.5l \\ -7.5l^2 & 0 & 2l^2 & -7.5l^2 \\ 0 & -6.5l & -7.5l^2 & l^2 \end{bmatrix} + \frac{8EI}{l^3} \begin{bmatrix} l^2 & -3l & .5l^2 & 0 \\ -3l & (24 + \frac{kl^3}{8EI}) & 0 & 3l \\ .5l^2 & 0 & 2l^2 & .5l^2 \\ 0 & 3l & .5l^2 & (l^2 + \frac{kl^3}{8EI}) \end{bmatrix} \right] \begin{Bmatrix} \theta_1 \\ \theta_2 \\ \theta_3 \\ \theta_4 \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \\ 0 \\ 0 \end{Bmatrix}$$

10-20 (a)



$$\begin{aligned}\varphi_1 &= 1 - 3\xi^2 + 2\xi^3 \\ \varphi_2 &= l\xi - l\xi^2 + l\xi^3 \\ \varphi_3 &= 3\xi^2 - 2\xi^3 \\ \varphi_4 &= -l\xi^2 + l\xi^3\end{aligned}$$

Element ①-② See Eq. (10.7-4)

Since $\bar{U}_1 = 0$ only φ_3 & φ_4 are needed.

$$\bar{F}_2 = \int_0^{l_1} p_0 \left(1 - \frac{\xi}{2}\right) (3\xi^2 - 2\xi^3) l_1 d\xi$$

$$M_2 = \int_0^{l_1} p_0 \left(1 - \frac{\xi}{2}\right) (-l\xi^2 + l\xi^3) l_1 d\xi$$

Integrated results.

$$\bar{F}_2 = .325 p_0 l_1$$

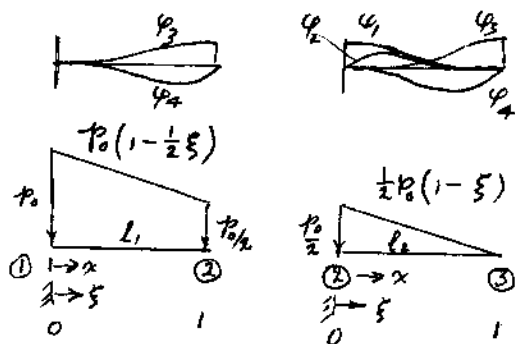
$$M_2 = -.0583 p_0 l_1^2$$

Assembly of both sections

$$\begin{Bmatrix} \bar{F}_2 \\ M_2 \\ \bar{F}_3 \\ M_3 \end{Bmatrix} = -p_0 \begin{Bmatrix} .325 l_1 + .250 l_2 \\ -.0583 l_1^2 - .850 l_2^2 \\ .0750 l_2 \\ .0583 l_2^2 \end{Bmatrix}$$

with $l_1 = l_2 = l/2$

$$\begin{Bmatrix} \bar{F}_2 \\ M_2 \\ \bar{F}_3 \\ M_3 \end{Bmatrix} = -p_0 l \begin{Bmatrix} .2875 \\ -.2271 \\ .0375 \\ .0146 \end{Bmatrix}$$



Element ②-③ See Eq. (10.7-4)

all 4 φ_s are necessary

$$\bar{F}_3 = \int_0^{l_2} \frac{p_0}{2} (1 - \xi) (3\xi^2 - 2\xi^3) l_2 d\xi$$

$$M_3 = \int_0^{l_2} \frac{p_0}{2} (1 - \xi) (-l_2 \xi^2 + l_2 \xi^3) l_2 d\xi$$

$$\bar{F}_2 = -\int_0^{l_2} \frac{p_0}{2} (1 - \xi) (1 - 3\xi^2 + 2\xi^3) l_2 d\xi$$

$$M_2 = -\int_0^{l_2} \frac{p_0}{2} (1 - \xi) (l_2 \xi - 2l_2 \xi^2 + l_2 \xi^3) l_2 d\xi$$

Integrated results

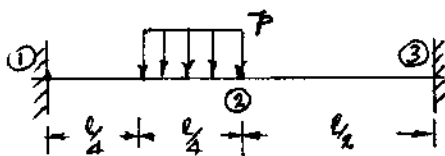
$$\bar{F}_2 = .25 p_0 l_2$$

$$M_2 = -.850 p_0 l_2^2$$

$$\bar{F}_3 = .0750 p_0 l_2$$

$$M_3 = .0583 p_0 l_2^2$$

10-20(b)



$$\begin{aligned}\bar{F}_2 &= -p \int_{\frac{l}{2}}^l \varphi_3 l d\xi = -p \int_{\frac{l}{2}}^l (3\xi^2 - 2\xi^3) l d\xi = -pl \left[3\left(\frac{\xi^3}{3}\right) - 2\left(\frac{\xi^4}{4}\right) \right]_{\frac{l}{2}}^l \\ &= -p [.4063] l_1 = -pl [.2031] \quad l_1 = \frac{l}{2}\end{aligned}$$

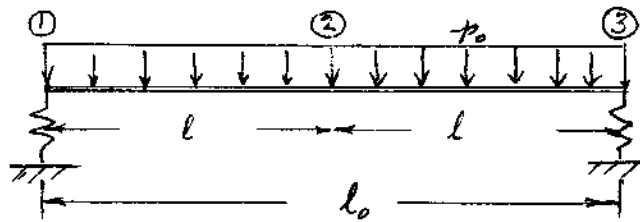
$$M_2 = -p \int_{\frac{l}{2}}^l (-l\xi^2 + l\xi^3) l d\xi = .0286 pl^2$$

with $\bar{u}_1 = \bar{u}_3 = \theta_1 = \theta_3 = 0$ (see Ex. 10.6-1)

$$\begin{Bmatrix} -P_2 \\ M_2 \end{Bmatrix} = \frac{EI}{l^3} \begin{bmatrix} 192 \\ 16l^2 \end{bmatrix} \begin{Bmatrix} \bar{u}_2 \\ \theta_2 \end{Bmatrix} = -pl \begin{Bmatrix} .2031 \\ .0286l \end{Bmatrix}$$

$$\therefore \bar{u}_2 = - \frac{.2031}{192} \frac{pl^4}{EI}$$

$$\theta_2 = - \frac{.0286}{16} \frac{pl^3}{EI}$$



Element $\begin{pmatrix} 1-2 \\ 2-3 \end{pmatrix}$ Eq. (10.2-1) for \bar{k} and Eq. (10.2-10) for \bar{m}

$$\bar{k} = \frac{EI}{l^3} \begin{bmatrix} 12 & 6l & -12 & 6l \\ 6l & 4l^2 & -6l & 2l^2 \\ -12 & -6l & 12 & -6l \\ 6l & 2l^2 & -6l & 4l^2 \end{bmatrix} \quad \bar{m} = \frac{ml}{420} \begin{bmatrix} 156 & 22l & 54 & -13l \\ 22l & 4l^2 & 13l & -3l^2 \\ 54 & 13l & 156 & -22l \\ -13l & -3l^2 & -22l & 4l^2 \end{bmatrix}$$

Assembled

$$\bar{k} = \frac{EI}{l^3} \begin{bmatrix} 12 & 6l & -12 & 6l \\ 6l & 4l^2 & -6l & 2l^2 \\ -12 & -6l & 12 & -6l \\ 6l & 2l^2 & -6l & 4l^2 \end{bmatrix} \begin{bmatrix} \bar{u}_1 \\ \bar{\theta}_1 \\ \bar{u}_2 \\ \bar{\theta}_2 \end{bmatrix}$$

must replace l by $\frac{l_0}{2}$

$$\bar{m} = \frac{ml}{420} \begin{bmatrix} 156 & 22l & 54 & -13l \\ 22l & 4l^2 & 13l & -3l^2 \\ 54 & 13l & 156 & -22l \\ -13l & -3l^2 & -22l & 4l^2 \end{bmatrix}$$

For loads, must determine generalized forces & moments at stations

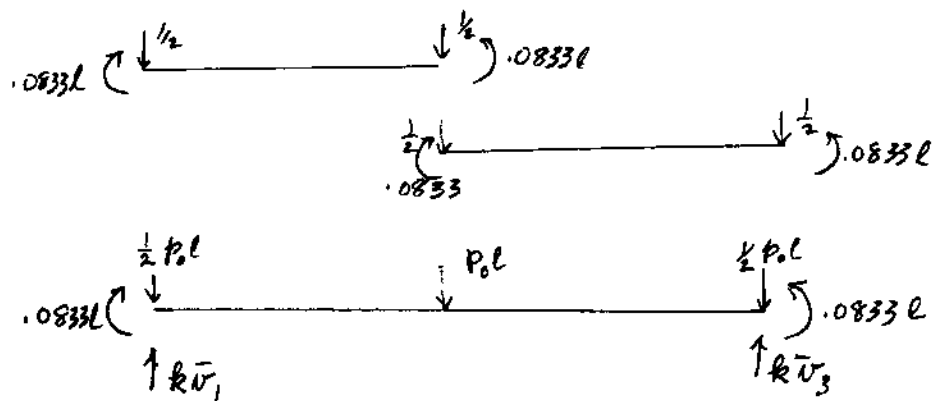
10-21 Cont.

$$\bar{F}_1 = -p_0 f(t) \int_0^1 \varphi_1 l d\xi = -p_0 f(t) \left[1 - 1 + \frac{1}{2} \right] l = -\frac{1}{2} p_0 l f(t)$$

$$\bar{M}_1 = -p_0 f(t) l \int_0^1 \varphi_2 l d\xi = -p_0 l^2 f(t) \left[\frac{1}{2} - \frac{2}{3} + \frac{1}{4} \right] = -.0833 p_0 l^2 f(t)$$

$$\bar{F}_2 = -p_0 f(t) \int_0^1 \varphi_3 l d\xi = -p_0 l f(t) \left[1 - \frac{2}{4} \right] = -\frac{1}{2} p_0 l f(t)$$

$$\bar{M}_2 = -p_0 f(t) l \int_0^1 \varphi_4 l d\xi = -p_0 l^2 f(t) \left[-\frac{1}{3} + \frac{1}{4} \right] = .0833 p_0 l^2 f(t)$$



$$\begin{Bmatrix} \bar{F}_1 - k\bar{v}_1 \\ \bar{M}_1 \\ \bar{F}_2 \\ \bar{M}_2 \\ \bar{F}_3 - k\bar{v}_3 \\ \bar{M}_3 \end{Bmatrix} = \begin{Bmatrix} -.50 - k\bar{v}_1 \\ -.0833 l \\ -1.0 \\ 0 \\ -.50 - k\bar{v}_3 \\ .0833 l \end{Bmatrix} p_0 l f(t)$$

$-k\bar{v}_1$ and $-k\bar{v}_3$ must now be shifted to stiffness matrix as $+k\bar{v}_1$ and $+k\bar{v}_3$

10-21 Cont General equation of motion 6x6

$$\frac{ml}{420} \begin{bmatrix} 156 & 22l & 54 & -13l & 0 & 0 \\ 22l & 4l^2 & 13l & -3l^2 & 0 & 0 \\ 54 & 13l & 312 & 0 & 54 & -13l \\ -13l & -3l & 0 & 8l^2 & 13l & -3l^2 \\ 0 & 0 & 54 & 13l & 156 & -22l \\ 0 & 0 & -13l & -3l^2 & -22l & 4l^2 \end{bmatrix} \begin{Bmatrix} \ddot{u}_1 \\ \ddot{\theta}_1 \\ \ddot{u}_2 \\ \ddot{\theta}_2 \\ \ddot{u}_3 \\ \ddot{\theta}_3 \end{Bmatrix} + \frac{EI}{l^3} \begin{bmatrix} (12+\frac{6l^3}{EI})6l & -12 & 6l & 0 & 0 \\ 6l & 4l^2 & -12 & 2l^2 & 0 & 0 \\ -12 & -6l & 24 & 0 & -12 & 6l \\ 6l & 2l^2 & 0 & 8l^2 & -6l & 2l^2 \\ 0 & 0 & -12 & -6l & (12+\frac{6l^3}{EI})-6l \\ 0 & 0 & 6l & 2l^2 & -6l & 4l^2 \end{bmatrix} \begin{Bmatrix} \bar{u}_1 \\ \theta_1 \\ \bar{u}_2 \\ \theta_2 \\ \bar{u}_3 \\ \theta_3 \end{Bmatrix} = \begin{Bmatrix} -50 \\ -0.033l \\ -1.0 \\ 0 \\ -50 \\ 0.033l \end{Bmatrix} \text{ (in ft)}$$

10-22

Symmetric mode (free-free vibration)

$$\bar{u}_1 = \bar{u}_3, \quad \theta_2 = 0, \quad \theta_1 = -\theta_3$$

Due to symmetry only \bar{u}_1 , θ_1 , and \bar{u}_2 are necessary

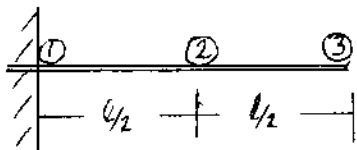
From above equation we have

$$\frac{ml}{420} \begin{bmatrix} 156 & 22l & 54 \\ 22l & 4l^2 & 13l \\ 54 & 13l & 312 \end{bmatrix} \begin{Bmatrix} \ddot{u}_1 \\ \ddot{\theta}_1 \\ \ddot{u}_2 \end{Bmatrix} + \frac{EI}{l^3} \begin{bmatrix} (12+\frac{6l^3}{EI})6l & -12 \\ 6l & 4l^2 & -6l \\ -12 & -6l & 24 \end{bmatrix} \begin{Bmatrix} \bar{u}_1 \\ \theta_1 \\ \bar{u}_2 \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \\ 0 \end{Bmatrix}$$

10-23 Problem deals with uniform cantilever equation in Ex. 10.5-1 whose equation is

$$-\lambda \begin{bmatrix} 312 & 0 & 54 & -6.5l \\ 0 & 2l^2 & 6.5l & -7.5l^2 \\ 54 & 6.5l & 156 & -11l \\ -6.5l & -7.5l^2 & -11l & l^2 \end{bmatrix} + \begin{bmatrix} 24 & 0 & -12 & 3l \\ 0 & 2l^2 & -3l & .5l^2 \\ -12 & -3l & 12 & -3l \\ 3l & .5l^2 & -3l & l^2 \end{bmatrix} \begin{Bmatrix} \bar{v}_2 \\ \theta_2 \\ \bar{v}_3 \\ \theta_3 \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \\ 0 \\ 0 \end{Bmatrix}$$

Numerical value of l is needed



Let $l = 20''$

$$\lambda = \left(\frac{\omega^2 m l}{840} \right) \left(\frac{l^3}{8EI} \right) = \frac{\omega^2 m l^3}{6720 EI}$$

Then

$$M = \begin{bmatrix} 320 & 0 & 54 & -130 \\ 0 & 800 & 130 & -300 \\ 54 & 130 & 156 & -220 \\ -130 & -300 & -220 & 400 \end{bmatrix}$$

$$K = \begin{bmatrix} 24 & 0 & -12 & 60 \\ 0 & 800 & -60 & 200 \\ -12 & -60 & 12 & -60 \\ 60 & 200 & -60 & 400 \end{bmatrix}$$

Computation was carried by two different computer programs. For the program CAL (which is somewhat similar to NROOT in that only M and K needed to be inputted) the eigenvalues and eigenvectors were printed out. The mode shapes are for the original displacement X (see computational notes)

10-23 Cont.Computer program CAL.

LOAD M R=4 C=4

ARRAY NAME = M

P M

NUMBER OF ROWS = 4 NUMBER OF COLUMNS = 4

COL#	=	1	2	3	4
ROW 1		320.00000	.00000	54.00000	-130.00000
ROW 2		.00000	800.00000	130.00000	-300.00000
ROW 3		54.00000	130.00000	156.00000	-220.00000
ROW 4		-130.00000	-300.00000	-220.00000	400.00000

LOAD K R=4 C=4

ARRAY NAME = K

P K

NUMBER OF ROWS = 4 NUMBER OF COLUMNS = 4

COL#	=	1	2	3	4
ROW 1		24.00000	.00000	-12.00000	60.00000
ROW 2		.00000	800.00000	-60.00000	200.00000
ROW 3		-12.00000	-60.00000	12.00000	-60.00000
ROW 4		60.00000	200.00000	-60.00000	400.00000

JACOBI K V M E

P V

Eigenvectors

COL#	=	1	2	3	4
ROW 1	$\begin{Bmatrix} \bar{u}_2 \\ \theta_2 \end{Bmatrix}$.02341	-.04958	-.00769	.03200
ROW 2	$\begin{Bmatrix} \bar{u}_2 \\ \theta_2 \end{Bmatrix}$.00401	.00154	.02962	.03379
ROW 3	$\begin{Bmatrix} \bar{u}_2 \\ \theta_2 \end{Bmatrix}$.06891	.06961	-.07766	.12991
ROW 4	$\begin{Bmatrix} \bar{u}_2 \\ \theta_2 \end{Bmatrix}$.00474	.01663	-.03745	.12536

COL#	=	1
ROW 1		.00183
ROW 2		.07203
ROW 3		.84016
ROW 4		7.02150

Eigen values = λ

$$\omega^2 = 6720 \frac{EI}{Ml^3} \lambda$$

$$\omega \left. \begin{matrix} 3.507 \\ 22.001 \\ 75.139 \\ 217.220 \end{matrix} \right\} \times \sqrt{\frac{EI}{Ml^3}}$$

Computational Notes

The matrix equation for the normal mode vibration is generally written as

$$[-M\lambda + K] X = 0 \quad (1)$$

where M and K are both symmetric matrices and λ is related to the natural frequencies by $\lambda = \omega^2$. Premultiplying the above equation by M^{-1} , we have another form of the equation

$$[-\lambda I + A] X = 0 \quad (2)$$

where

$$A = M^{-1}K \quad (3)$$

is called the dynamic matrix. In general $M^{-1}K$ is not symmetric.

It is desirable to introduce a coordinate transformation to rewrite the above equation in the standard form incorporated into most computer programs. This standard form is

$$[-\lambda I + \tilde{A}] U = 0 \quad (4)$$

where I is a unit matrix and \tilde{A} is a symmetric matrix.

For this transformation, let

$$X = Q^{-1}U \quad (5)$$

and substitute into Eq. (1).

$$[-\lambda M Q^{-1} + K Q^{-1}] U = 0$$

Premultiply by the transpose of Q which we will designate by Q^{-T}

$$[-\lambda Q^{-T} M Q^{-1} + Q^{-T} K Q^{-1}] U = 0$$

If we now decompose M as

$$M = Q^T Q \quad (6)$$

the first term of the last equation becomes a unit matrix.

$$\text{i.e.} \quad Q^{-T} Q^T Q Q^{-1} = I$$

We note here that

$$M = Q^T Q = M^{\frac{1}{2}} M^{\frac{1}{2}}$$

and since M is symmetric, we obtain

$$Q = Q^T = M^{\frac{1}{2}} \quad (7)$$

and

$$\tilde{A} = Q^{-T} K Q^{-1} = M^{-\frac{1}{2}} K M^{-\frac{1}{2}} \quad (8)$$

which is symmetric. Thus the standard form of the computer equation for the eigenvalues and eigenvectors is achieved.

The eigenvalues, λ , of Eq.(4) will be identical to those of Eqs.(1) or (2). However, the eigenvectors of the computer equation, Eq.(4), will now be U instead of X. To obtain the eigenvectors of the original equation, the transformation equation, Eq.(5), must be used.

For lumped mass systems, where the coordinates are chosen at each mass, the mass matrix is diagonal and $Q = M^{1/2}$ is easily determined. However when M is a full matrix, as in most finite element formulations, the determination of $M^{1/2}$ is not a simple task. For such cases, $M^{1/2}$ is determined from the equation

$$M^{\frac{1}{2}} = Y \Lambda^{\frac{1}{2}} Y^T \quad (9)$$

where Λ is the diagonal matrix of the eigenvalues of M and Y is the orthonormal matrix of the eigenvectors of M,

i.e. $Y^T Y = I$. (M must here be a positive definite real symmetric matrix, which the mass matrix satisfies. See Meirovitch, L. Analytical Methods in Vibration, p 76, The Macmillan Co., N.Y., 1967)

Ref: Meirovitch, L. Computational Methods in Structural Dynamics, p61, Sijthoff & Noordhoff, The Netherlands, 1980.

10-23 Cont. (2nd Procedure) The steps in Computational Notes were carried out on MacPlus with BASIC programs in Math Package and Scientific Analysis Program SAP

$$\text{Computation for } M^{\frac{1}{2}} = Y \Lambda^{\frac{1}{2}} Y^T$$

INPUT MATRIX M IN SYMMETRIC FORM

$$\begin{bmatrix} 320 & & & \\ 0 & 800 & & \\ 54 & 130 & 156 & \\ -130 & -300 & -220 & 400 \end{bmatrix} = M$$

EIGENVALUES OF M (by SYMEIG)

1026.2458250538
427.64188324099
201.17548671333
20.936804991822

$$\Lambda = \begin{bmatrix} 1026.24 & & & \\ & 427.64 & & \\ & & 201.17 & \\ & & & 20.93 \end{bmatrix}$$

CORRESPONDING EIGENVECTORS (COLS) OF M

$$\begin{bmatrix} -.11251123964196 & .69783094122845 & -.69987273057571 & .10271981023802 \\ -.81659856817195 & -.46966360076447 & -.32448266951864 & .08540420223171 \\ -.25654886082296 & .28577999506147 & .44493141685562 & .80904172372534 \\ .50466888295643 & -.45910551510988 & -.45489030817742 & .57236898247361 \end{bmatrix} = Y$$

$$\left(\begin{matrix} -.112511 & -.816598 & -.256548 & .5046688 \end{matrix} \right) \begin{Bmatrix} -.112511 \\ -.816598 \\ -.256548 \\ .5046688 \end{Bmatrix}$$

$$= .01266 + .666832 + .065817 + .25491 = 1.0000$$

The Y_s are orthonormal as shown above checking

$$\Lambda^{\frac{1}{2}} = \begin{bmatrix} 32.051 & 0 & 0 & 0 \\ 0 & 20.679 & 0 & 0 \\ 0 & 0 & 14.184 & 0 \\ 0 & 0 & 0 & 4.576 \end{bmatrix}$$

Computation for $Y\Lambda^{\frac{1}{2}}Y^P = M^{\frac{1}{2}}$

THE PRODUCT MATRIX [C] = [A] X [B] IS : -

-3.605737 14.42981 -9.925963 .4699552
-26.17285 -9.7121 -4.602424 .3907904
-7.900571 5.90799 6.310462 3.701984
16.17293 -9.49373 -6.450883 2.618845

} $Y\Lambda^{\frac{1}{2}}$

THE PRODUCT MATRIX [C] = [A] X [B] IS : -

17.46832 -.5734422 1.011564 -3.660839
-.5723705 27.45703 2.207047 -6.432595
.9751865 1.94517 9.515408 -7.450424
-3.661047 -6.428366 -7.611399 16.95033

} $Y\Lambda^{\frac{1}{2}}Y^T = M^{\frac{1}{2}} = Q$

The inverse of the input matrix is : -

0.0608 0.0050 0.0068 0.0180
0.0050 0.0405 0.0050 0.0186
0.0070 0.0063 0.1635 0.0758
0.0181 0.0193 0.0768 0.1040

= Q^{-1}

Multiplication for $\bar{Q}^T K \bar{Q}^{-1}$
by SAP (Matrix)

THE PRODUCT MATRIX [C] = [A] X [B] IS : -

2.4612 7.2 -2.0316 11.468
1.2024 35.882 -3.5724 15.742
2.8092 9.550001 -3.0276 22.318
5.7624 31.132 -6.6624 41.852

$\bar{Q}^T K$

THE PRODUCT MATRIX [C] = [A] X [B] IS : -

.3789906 .5124393 .601312 1.216898
.5124393 1.740547 .8124846 2.055429
.601312 .8124846 1.285862 2.319776
1.216898 2.055429 2.319776 4.530376

$\bar{Q}^T K \bar{Q}^{-1} = \tilde{A}$

10-23 Cont.

INPUT MATRIX \tilde{A} IN SYMMETRIC FORM

.37899
.51243 1.7405
.601312 .81248 1.2858
1.2168 2.0554 2.3197 4.5303

Eigen values & vectors
by Math Package (SYMEIG)

EIGENVALUES OF \tilde{A}

7.0278455556106
.83378223012746
.072052235090057
.0019099791723867

$$\lambda = \frac{\tilde{\omega}^2 M L^3}{6720 EI}$$

$$\tilde{\omega}^2 = \frac{6720 \cdot EI}{M L^3} \lambda$$

47226.
5603.
484.
12.83

ω		
computed	exact	
217.315		$\left. \begin{matrix} \\ \\ \\ \end{matrix} \right\} \times \frac{EI}{ML^3}$
74.853		
22.000		
3.5825		
		$\omega_3 = 61.7$
		$\omega_2 = 22.03$
		$\omega_1 = 3.516$

CORRESPONDING EIGENVECTORS (COLS) OF $\tilde{A} = U$

.21289484519686 .092870186818959 -.856536608263 .46086435313892
.39308961378661 -.88527738654973 .11908614674759 .21813526548855
.40095055035245 .4060306420499 .49588307863158 .65458211571615
.79962178346634 .20687724616463 -.079142281272342 -.55816064689359

must change to
 $X = Q^T U$

use SAP(MAT)

Original eigenvectors X

$$X = Q^T U$$

THE PRODUCT MATRIX $[C] = [A] X [B]$ IS :-

3.202216E-02 7.699439E-03 -4.953256E-02 2.351194E-02
3.385756E-02 -2.951012E-02 1.54474E-03 4.028889E-03
.1301223 7.712928E-02 6.982172E-02 .0693064
.1253841 3.728332E-02 1.664509E-02 4.773013E-03

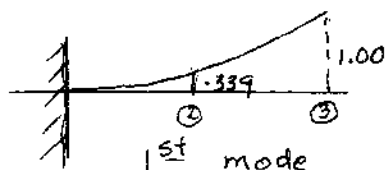
$$\left. \begin{matrix} \\ \\ \\ \end{matrix} \right\} = \begin{matrix} \tilde{u}_2 \\ \theta_2 \\ \tilde{u}_3 \\ \theta_3 \end{matrix}$$

4th mode 3rd mode 2nd mode 1st mode

Checking mode shapes (for 1st mode)

$$\left(\frac{\tilde{u}_2}{\tilde{u}_3} \right)_1 = \frac{.0235}{.0693} = .339$$

$$\left(\frac{\theta_2}{\theta_3} \right) = \frac{.004029}{.004773} = .844$$



11-1

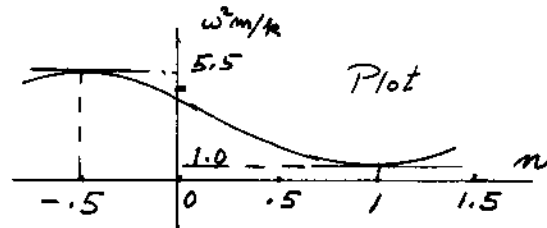
$$T = \frac{1}{2} m \dot{x}_1^2 + \frac{1}{2} (2m) \dot{x}_2^2 \quad U = \frac{1}{2} k x_1^2 + \frac{1}{2} (3k) (x_2 - x_1)^2 + \frac{1}{2} (2k) x_2^2$$

Let $\dot{x} = \omega x$ & equate

$$\omega^2 m x_1^2 + 2\omega^2 m x_2^2 = k x_1^2 + 3k (x_2 - x_1)^2 + 2k x_2^2$$

Let $x_2/x_1 = n$, then

$$\frac{\omega^2 m}{k} = \frac{1 + 3(n-1)^2 + 2n^2}{1 + 2n^2}$$

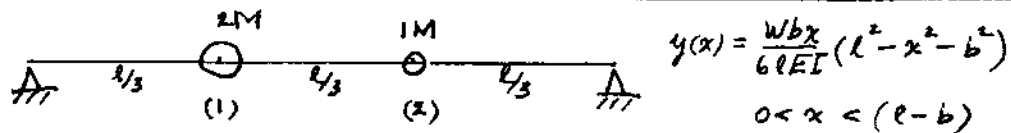


Can be checked by

$$\frac{\partial}{\partial n} \left(\frac{\omega^2 m}{k} \right) = 0 \rightarrow \text{gives } n = -0.5 \text{ and } n = 1.0$$

$$\text{Roots are } \frac{\omega^2 m}{k} = 1.0 \text{ and } \frac{\omega^2 m}{k} = 5.50$$

11-2



Due to 2M

$$y_1 = \frac{(2Mg) \frac{2}{3}l \frac{1}{3}l}{6EI} \left[l^2 - \left(\frac{l}{3}\right)^2 - \left(\frac{2l}{3}\right)^2 \right] = \frac{16}{486} \left(\frac{Mgl^3}{EI} \right)$$

$$y_2 = \frac{(2Mg) \frac{1}{3}l \frac{1}{3}l}{6EI} \left[l^2 - \left(\frac{l}{3}\right)^2 - \left(\frac{l}{3}\right)^2 \right] = \frac{14}{486} \left(\frac{Mgl^3}{EI} \right)$$

Due to 1M

$$y_1 = \frac{Mg \frac{1}{3}l \frac{1}{3}l}{6EI} \left[l^2 - \left(\frac{l}{3}\right)^2 - \left(\frac{l}{3}\right)^2 \right] = \frac{7}{486} \left(\frac{Mgl^3}{EI} \right)$$

$$y_2 = \frac{Mg \frac{1}{3}l \frac{2}{3}l}{6EI} \left[l^2 - \left(\frac{2l}{3}\right)^2 - \left(\frac{l}{3}\right)^2 \right] = \frac{8}{486} \left(\frac{Mgl^3}{EI} \right)$$

$$\text{Add } y_1 = \frac{23}{486} \left(\frac{Mgl^3}{EI} \right) \quad y_2 = \frac{22}{486} \left(\frac{Mgl^3}{EI} \right)$$

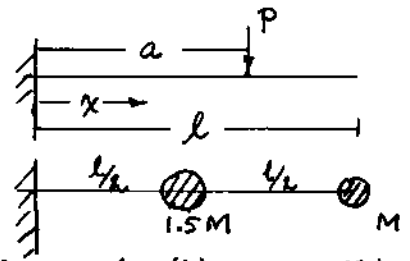
$$\omega_1^2 = \frac{g \left[(2M) \frac{23}{486} + (M) \frac{22}{486} \right]}{\left[2M \left(\frac{23}{486} \right)^2 + M \left(\frac{22}{486} \right)^2 \right]} \frac{1}{\frac{Mgl^3}{EI}} = 21.43 \frac{EI}{Ml^3}$$

$$\omega_1 = 4.63 \sqrt{\frac{EI}{Ml^3}} \text{ rad/s.}$$

11-3

$$y(x) = \frac{Px^2}{6EI} (3a-x) \quad 0 \leq x \leq a$$

$$= \frac{Pa^2}{6EI} (3x-a) \quad a \leq x \leq l$$



Due to $1.5M$ & M with results superimposed (1) (2)

$$y_1 = \frac{1.5Mg(\frac{l}{2})^2}{3EI} + \frac{Mg(\frac{l}{2})^2}{6EI} (3l - \frac{l}{2}) = \frac{16}{96} \frac{Mgl^3}{EI}$$

$$y_2 = \frac{1.5Mg(\frac{l}{2})^2}{6EI} (3l - \frac{l}{2}) + \frac{Mgl^3}{3EI} = \frac{47}{96} \frac{Mgl^3}{EI}$$

$$\omega^2 = \frac{[1.5(\frac{16}{96}) + (\frac{47}{96})] \frac{EI}{Ml^3}}{[1.5(\frac{16}{96})^2 + (\frac{47}{96})^2]} = 2.628 \frac{EI}{Ml^3}$$

$$\therefore \omega_1 = 1.621 \sqrt{\frac{EI}{Ml^3}}$$

11-4

$$a_{11} = \frac{1}{24} \frac{l^3}{EI}$$

$$a_{12} = a_{21} = \frac{10}{96} \frac{l^3}{EI}$$

$$a_{22} = \frac{l^3}{3EI}$$

$$\therefore [a] = \frac{l^3}{96EI} \begin{bmatrix} 4 & 10 \\ 10 & 32 \end{bmatrix}$$

$$[k] = [a]^{-1} = \frac{96EI}{l^3} \frac{1}{28} \begin{bmatrix} 32 & -10 \\ -10 & 4 \end{bmatrix} = 13.7143 \frac{EI}{l^3} \begin{bmatrix} 8 & -2.5 \\ -2.5 & 1 \end{bmatrix}$$

Set up eq. of motion using flexibility

$$\begin{Bmatrix} X_1 \\ X_2 \end{Bmatrix} = \frac{\omega^2 M l^3}{96 EI} \begin{bmatrix} 4 & 10 \\ 10 & 32 \end{bmatrix} \begin{bmatrix} 1.5 & 0 \\ 0 & 1.0 \end{bmatrix} \begin{Bmatrix} X_1 \\ X_2 \end{Bmatrix}$$

$$= \frac{\omega^2 M l^3}{96 EI} \begin{bmatrix} 6 & 10 \\ 15 & 32 \end{bmatrix} \begin{Bmatrix} X_1 \\ X_2 \end{Bmatrix} \quad \text{let } \lambda = \frac{\omega^2 M l^3}{96 EI}$$

$$\begin{bmatrix} (1-6\lambda) & -10\lambda \\ -15\lambda & (1-32\lambda) \end{bmatrix} \begin{Bmatrix} X_1 \\ X_2 \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \end{Bmatrix}$$

11-4 Cont.

$$42\lambda^2 - 38\lambda + 1 = 0$$

$$\lambda^2 - 0.9048\lambda + 0.0238 = 0$$

$$\lambda = 0.4524 \pm \sqrt{.2048 - .0238} = 0.4524 \pm 0.4253$$

$$\lambda = \frac{\omega^2 M L^3}{96EI} = \begin{cases} 0.0271 \\ 0.8771 \end{cases} \quad \omega^2 = \begin{cases} 2.6016 \frac{EI}{ML^3} \\ 84.2576 \frac{EI}{ML^3} \end{cases}$$

$$\omega = \begin{cases} 1.6129 \sqrt{\frac{EI}{ML^3}} \\ 9.1792 \sqrt{\frac{EI}{ML^3}} \end{cases} \left(\frac{X_1}{X_2} \right) = \frac{10\lambda}{1-6\lambda} = 0.3237$$

To verify $\omega^2 = \frac{X' K X}{X' M X}$

$$\omega^2 = \frac{(.3237 \ 1.0) \begin{bmatrix} 8 & -2.5 \\ -2.5 & 1 \end{bmatrix} \begin{Bmatrix} .3237 \\ 1.0 \end{Bmatrix}}{(.3237 \ 1.0) \begin{bmatrix} 1.5 & 0 \\ 0 & 1 \end{bmatrix} \begin{Bmatrix} .3237 \\ 1.0 \end{Bmatrix}} = \frac{13.7143 \frac{EI}{L^3}}{2.605 \frac{EI}{ML^3}} = 2.605 \frac{EI}{ML^3}$$

$$\therefore \omega_1 = 1.614 \sqrt{\frac{EI}{ML^3}}$$

11-5

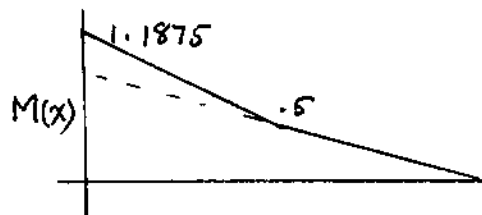
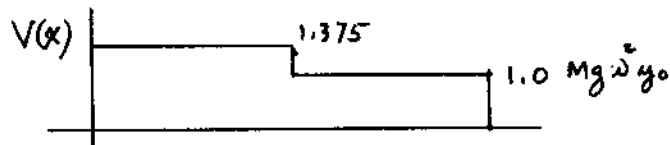
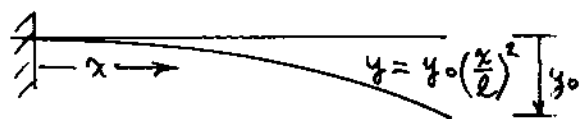
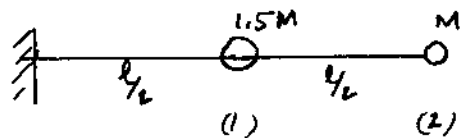
$$\omega^2 = \frac{X' M X}{X' M' a M X} =$$

$$= \frac{(.3237 \ 1.0) \begin{bmatrix} 1.5 & 0 \\ 0 & 1 \end{bmatrix} \begin{Bmatrix} .3237 \\ 1.0 \end{Bmatrix} \cdot \frac{96EI}{ML^3}}{(.3237 \ 1.0) \begin{bmatrix} 1.5 & 0 \\ 0 & 1 \end{bmatrix} \begin{bmatrix} 4 & 10 \\ 0 & 32 \end{bmatrix} \begin{bmatrix} 1.5 & 0 \\ 0 & 1 \end{bmatrix} \begin{Bmatrix} .3237 \\ 1.0 \end{Bmatrix}}$$

$$= \frac{1.1572}{42.654} \cdot \frac{96EI}{ML^3} = 2.6045 \frac{EI}{ML^3}$$

$$\omega_1 = 1.6138 \sqrt{\frac{EI}{ML^3}}$$

11-6



$$\therefore V(x) = Mg \omega^2 y_0 \quad \frac{l}{2} \leq x \leq l$$

$$= 1.375 Mg \omega^2 y_0 \quad 0 \leq x \leq \frac{l}{2}$$

$$M(x) = \int_x^l V(\xi) d\xi = Mg \omega^2 y_0 \xi \Big|_x^l = Mg \omega^2 y_0 (l-x) \quad \frac{l}{2} \leq x \leq l$$

$$= Mg \omega^2 y_0 \left[\frac{l}{2} + 1.375 \left(\frac{l}{2} - x \right) \right] \quad 0 \leq x \leq \frac{l}{2}$$

$$U = \frac{1}{2} \int \frac{M^2}{EI} dx = \frac{(Mg \omega^2 y_0)^2}{2EI} \left[\int_{l/2}^l (l-x)^2 dx + \int_0^{l/2} \left\{ \frac{l}{2} + 1.375 \left(\frac{l}{2} - x \right) \right\}^2 dx \right]$$

$$= \frac{1}{2} \frac{(Mg \omega^2 y_0)^2}{EI} l^3 \left[\frac{1}{24} + \frac{9.0156}{24} \right] = \frac{1}{2} \frac{M^2 g^2 \omega^4 y_0^2}{EI} (0.4173)$$

$$T = \frac{1}{2} M \omega^2 y_0^2 + \frac{1}{2} (1.5M) \omega^2 y_0^2 \left(\frac{1}{2} \right)^2 = \frac{1}{2} M \omega^2 y_0^2 (1.0938)$$

$$\dot{y} = \omega y = \omega y_0 \left(\frac{x}{l} \right)^2$$

$$\ddot{y} = -\omega^2 y = -\omega^2 y_0 \left(\frac{x}{l} \right)^2$$

$$-m \ddot{y} = \omega^2 y \left(\frac{x}{l} \right)^2 m$$

For lumped mass, dynamic

loads are

$$\omega^2 M y_0 g \text{ at } (2)$$

$$\text{and } \omega^2 1.5 M y_0 g \left[\frac{1}{4} \right] +$$

$$\omega^2 M y_0 g [1]$$

$$\text{at } (1) = \omega^2 M g y_0 1.375$$

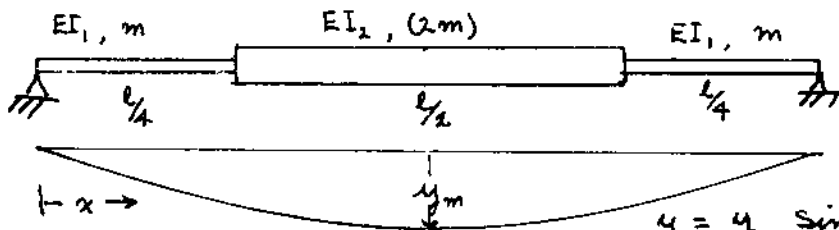
11-6 Cont

$$U = T \quad \text{gives}$$

$$\omega^2 = \frac{1.0938}{0.4173} \frac{EI}{Ml^3} = 2.619 \frac{EI}{Ml^3}$$

$$\omega_1 = 1.618 \sqrt{\frac{EI}{Ml^3}} \quad \text{Exact val} = 1.6129 \sqrt{\frac{EI}{Ml^3}}$$

11-7



$$y = y_m \sin \frac{\pi x}{l}$$

$$\frac{d^2 y}{dx^2} = -y_m \left(\frac{\pi}{l} \right)^2 \sin \frac{\pi x}{l}$$

$$U = \frac{1}{2} \int EI \left(\frac{d^2 y}{dx^2} \right)^2 dx = \frac{1}{2} y_m^2 \left(\frac{\pi}{l} \right)^4 \left[EI_1 \int_0^{l/4} \sin^2 \frac{\pi x}{l} dx + EI_2 \int_{l/4}^{l/2} \sin^2 \frac{\pi x}{l} dx \right]$$

$$= y_m^2 \left(\frac{\pi}{l} \right)^4 \left[EI_1 \left(\frac{1}{2} \left(x - \frac{l}{2\pi} \sin \frac{2\pi x}{l} \right) \right) \Big|_0^{l/4} + EI_2 \left(\frac{1}{2} \left(x - \frac{l}{2\pi} \sin \frac{2\pi x}{l} \right) \right) \Big|_{l/4}^{l/2} \right]$$

$$= y_m^2 \left(\frac{\pi}{l} \right)^4 l \left[.0454 EI_1 + .2046 EI_2 \right]$$

$$T = \frac{1}{2} \int m \dot{y}^2 dx = \frac{1}{2} \omega^2 y_m^2 \left[m \int_0^{l/4} \sin^2 \frac{\pi x}{l} dx + (2m) \int_{l/4}^{l/2} \sin^2 \frac{\pi x}{l} dx \right]$$

$$= \omega^2 y_m^2 m l \left[.0454 + .4092 \right] = 0.4546 \omega^2 y_m^2 m l$$

Equating

$$\omega_1^2 = \frac{\pi^4}{m l^4} \left[0.100 EI_1 + 0.450 EI_2 \right]$$

$$\omega_1 = \pi^2 \sqrt{\frac{0.100 EI_1 + 0.450 EI_2}{m l^4}}$$

11-7 Cont:

(a) if $EI_2 = EI_1$ & $m_1 = m$, $m_2 = 2m$

$$\omega_1 = .7416 \pi^2 \sqrt{\frac{EI_1}{m l^4}} = 7.32 \sqrt{\frac{EI_1}{m l^4}}$$

(b) if $EI_2 = 4EI_1$,

$$\omega_1 = 1.3785 \pi^2 \sqrt{\frac{EI_1}{m l^4}}$$

11-8

$$y = y_0 \frac{x}{l} \left(1 - \frac{x}{l}\right) \quad \frac{dy}{dx} = \frac{y_0}{l} \left(1 - 2 \frac{x}{l}\right)$$

$$\frac{d^2 y}{dx^2} = -y_0 \frac{2}{l^2}$$

$$U = \frac{1}{2} \frac{4 y_0^2}{l^4} \left[2EI_1 \int_0^{l/4} dx + 2EI_2 \int_{l/4}^{l/2} dx \right] = \frac{1}{2} \left(\frac{4 y_0^2}{l^4} \right) \frac{l}{2} [EI_1 + EI_2]$$

$$= \frac{y_0^2}{l^3} [EI_1 + EI_2]$$

$$T = \frac{1}{2} \omega^2 y_0^2 \left[m \int_0^{l/4} \left(\frac{x}{l}\right)^2 \left(1 - \frac{x}{l}\right)^2 dx + (2m) \int_{l/4}^{l/2} \left(\frac{x}{l}\right)^2 \left(1 - \frac{x}{l}\right)^2 dx \right]$$

$$= \omega^2 y_0^2 \left[ml \left\{ \frac{1}{3} \left(\frac{x}{l}\right)^3 - \frac{1}{2} \left(\frac{x}{l}\right)^4 + \frac{1}{5} \left(\frac{x}{l}\right)^5 \right\}_0^{l/4} + 2ml \left\{ \text{same} \right\}_{l/4}^{l/2} \right]$$

$$= \omega^2 y_0^2 ml \left[(.008659) + (.0264) \right] = .03506 \omega^2 y_0^2 ml$$

Equating $T = U$

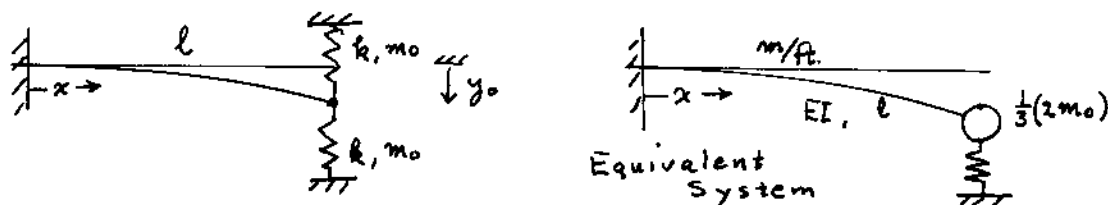
$$\omega_1^2 = 28.52 \left(\frac{EI_1 + EI_2}{m l^4} \right)$$

For $EI_2 = EI_1$, $\omega_1 = \sqrt{57.04} \sqrt{\frac{EI_1}{m l^4}} = 7.55 \sqrt{\frac{EI_1}{m l^4}}$

for Sine Curve Prob. 11-7

$$\omega_1 = 7.32 \sqrt{\frac{EI_1}{m l^4}}$$

11-9



$$\left. \begin{aligned} T_m &= \frac{1}{2} \left(\frac{33ml}{140} \right) \omega^2 y_0^2 + \frac{1}{2} \left(\frac{2}{3} m_0 \right) \omega^2 y_0^2 \\ U_m &= \frac{1}{2} \left(\frac{3EI}{l^3} \right) y_0^2 + \frac{1}{2} (2k) y_0^2 \end{aligned} \right\} \text{Equate}$$

$$\omega_1^2 = \frac{\frac{3EI}{l^3} + 2k}{\left(\frac{33ml}{140} + \frac{2}{3} m_0 \right)}$$

11-10

$$y_m = \sin \frac{\pi x}{l} + b$$

$$U = \frac{1}{2} EI \int_0^l (y'')^2 dx + \frac{1}{2} \left(\frac{k}{2} \right) y_{x=0}^2 + \frac{1}{2} \left(\frac{k}{2} \right) y_{x=l}^2$$

$$U_{max} = \frac{1}{2} EI \int_0^l \left(\frac{\pi}{l} \right)^4 \sin^2 \frac{\pi x}{l} dx + \frac{1}{2} k b^2$$

$$\begin{aligned} T_{max} &= \frac{1}{2} \int_0^l \frac{M}{l} \dot{y}^2 dx = \frac{1}{2} \omega^2 \frac{M}{l} \int_0^l \left(\sin^2 \frac{\pi x}{l} + 2b \sin \frac{\pi x}{l} + b^2 \right) dx \\ &= \frac{M\omega^2}{2} \left(\frac{1}{2} + \frac{4b}{\pi} + b^2 \right) \end{aligned}$$

$$\text{Equating } U_{max} = T_{max} \quad \omega_1^2 = \frac{2k}{M} \left(\frac{\frac{K}{2} \frac{\pi^4}{4} + \frac{b^2}{2}}{\frac{1}{2} + \frac{4b}{\pi} + b^2} \right)$$

$$\text{where } K = EI/l^3$$

$$\frac{\partial \omega_1^2}{\partial b} = 0 \quad \text{gives}$$

$$b = -\frac{\pi}{4} \left(\frac{1}{2} - \frac{K\pi^4}{2k} \right) \pm \sqrt{\left[\frac{\pi}{4} \left(\frac{1}{2} - \frac{K\pi^4}{2k} \right) \right]^2 + \frac{\pi^4 K}{2k}}$$

11-11

$$y = y_m \left[3\left(\frac{x}{l}\right) - 4\left(\frac{x}{l}\right)^3 \right] \quad 0 \leq x \leq \frac{l}{2}$$

$$m(x) = m_0 \left(\frac{x}{l}\right) \left(1 - \frac{x}{l}\right)$$

$$T_m = \frac{1}{2} \times 2 \int_0^{l/2} m(x) \omega^2 y^2 dx = m_0 \omega^2 y_m^2 \int_0^{l/2} \frac{x}{l} \left[9\left(\frac{x}{l}\right)^2 - 24\left(\frac{x}{l}\right)^4 + 16\left(\frac{x}{l}\right)^6 \right] dx$$

$$= m_0 \omega^2 y_m^2 l \left[\frac{9}{4} \left(\frac{1}{2}\right)^4 - \frac{24}{6} \left(\frac{1}{2}\right)^6 + \frac{16}{8} \left(\frac{1}{2}\right)^8 - \frac{9}{5} \left(\frac{1}{2}\right)^5 + \frac{24}{7} \left(\frac{1}{2}\right)^7 - \frac{16}{9} \left(\frac{1}{2}\right)^9 \right]$$

$$= 0.0529 m_0 l \omega^2 y_m^2$$

$$U_m = \frac{1}{2} \times 2 \int_0^{l/2} EI \left(\frac{d^2 y}{dx^2} \right)^2 dx = y_m^2 EI \int_0^{l/2} \frac{24^2}{l^6} x^2 dx = y_m^2 EI \frac{24^2}{l^6} \frac{1}{3} \left(\frac{l}{2}\right)^3$$

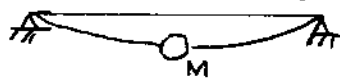
Equating

$$\omega_1^2 = 453 \frac{EI}{(m_0 l) l^3}$$

$$\omega_1 = 21.30 \sqrt{\frac{EI}{(m_0 l) l^3}}$$

To check, first find total mass

$$\int m dx = 2m_0 \int_0^{l/2} \left(\frac{x}{l} - \frac{x^3}{l^3} \right) dx = 2m_0 \left[\frac{x^2}{2l} - \frac{x^4}{4l^3} \right]_0^{l/2} = 0.2188 m_0 l = M_T$$

 For massless beam with M at midspan $k = \frac{48EI}{l^3}$

$$T = \frac{1}{2} m_{eff} \omega^2 y_m^2 = 0.0529 m_0 l \omega^2 y_m^2 \text{ from above}$$

$$\therefore m_{eff} = .1058 m_0 l = N M_T = N .2188 m_0 l$$

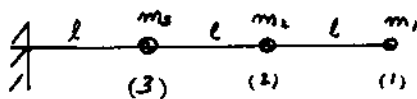
$$\therefore N = .4835 \quad \therefore m_{eff} = .4835 (\text{total mass})$$

$$\omega^2 = \frac{k}{m_{eff}} = \frac{48EI}{.4835 M_T l^3} = 99.28 \frac{EI}{M_T l^3}$$

$$\omega = 9.96 \sqrt{\frac{EI}{M_T l^3}} \quad \therefore \text{results appear to be reasonable}$$

11-12

From Eq. 11.2-3 $\frac{1}{\omega_1^2} \cong a_{11} m_1 + a_{22} m_2 + a_{33} m_3$



$$a_{11} = \frac{(3l)^3}{3EI} = \frac{27l^3}{3EI}$$

$$a_{22} = \frac{(2l)^3}{3EI} = \frac{8l^3}{3EI}$$

$$a_{33} = \frac{l^3}{3EI}$$

$$\therefore \frac{1}{\omega_1^2} \cong (27m_1 + 8m_2 + m_3) \frac{l^3}{3EI} \quad \omega_1 \cong \sqrt{\frac{3EI}{l^3} \left(\frac{1}{27m_1 + 8m_2 + m_3} \right)}$$

11-13

Flex. $a_{xb} = \frac{bx}{6EI} (l^2 - x^2 - b^2)$

$$a_{11} = \frac{\frac{3}{4}l \cdot \frac{1}{4}l}{6EI} (l^2 - \frac{l^2}{16} - \frac{9}{16}l^2) = \frac{3}{16^2} \frac{l^3}{EI} = a_{33}$$

$$a_{22} = \frac{\frac{1}{2}l \cdot \frac{1}{2}l}{6EI} (l^2 - \frac{l^2}{4} - \frac{l^2}{4}) = \frac{1}{48} \frac{l^3}{EI} = \frac{16}{3} \cdot \frac{l^3}{16^2 EI}$$

$$\frac{1}{\omega_1^2} \cong a_{11} \frac{W_1}{g} + a_{22} \frac{W_2}{g} + a_{33} \frac{W_3}{g}$$

$$= \frac{l^3}{16^2 EI g} \left(3W_1 + \frac{16}{3} W_2 + 3W_3 \right) = \frac{l^3 W}{16^2 EI} \left(3 + \frac{16}{3} \times 4 + 6 \right)$$

$$\therefore \omega_1 \cong 16 \sqrt{\frac{3}{91} \frac{EI g}{W l^3}} = 2.905 \sqrt{\frac{EI g}{W l^3}}$$

11-14

Use Eq. (b) Ex 11.2-1 with $a_{22} = \frac{0.78}{1000}$

$$\frac{1}{\omega_1^2} \cong \frac{1}{\omega_{11}^2} + \frac{0.78}{1000} \frac{320}{386} = \left(\frac{60}{2\pi \times 622} \right)^2 + 0.6466 \times 10^{-3}$$

$$= (0.2357 + 0.6466) \times 10^{-3}$$

$$= 0.8823 \times 10^{-3}$$

$$\omega_1 \cong \sqrt{\frac{1}{0.8823 \times 10^{-3}}} = 33.66 \text{ rad/sec} = 5.358 \text{ Hz.}$$

11-15

$$\frac{1}{(\omega_1)_1^2} - \frac{1}{\omega_{11}^2} = a_{22}(m_2)_1$$

$$\frac{1}{(\omega_1)_2^2} - \frac{1}{\omega_{11}^2} = a_{22}(m_2)_2 \quad \text{divide to eliminate } a_{22}$$

$$\frac{\omega_{11}^2 - (\omega_1)_1^2}{(\omega_1)_1^2} \cdot \frac{(\omega_1)_2^2}{\omega_{11}^2 - (\omega_1)_2^2} = \frac{(m_2)_1}{(m_2)_2}$$

$$(\omega_1)_1 = 435 \times 2\pi \quad (m_2)_1 = 5.44$$

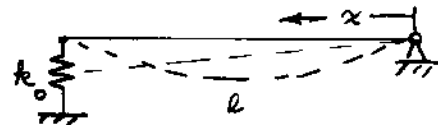
$$(\omega_1)_2 = 398 \times 2\pi \quad (m_2)_2 = 5.44 + 4.52 = 9.96$$

$$\left(\frac{398}{435}\right)^2 \left[\frac{f_{11}^2 - 435^2}{f_{11}^2 - 398^2} \right] = \frac{5.44}{9.96} \quad \therefore f_{11}^2 = 245219$$

$$f_{11} = 495.2 \text{ cps}$$

11-16

$$\phi_1 = \frac{x}{l}, \quad \phi_2 = \sin \frac{\pi x}{l}$$



$$m_{11} = \int_0^l m \phi_1 \phi_1 dx = \frac{m}{l^2} \int_0^l x^2 dx = \frac{ml}{3}$$

$$m_{12} = \int_0^l m \frac{x}{l} \cdot \sin \frac{\pi x}{l} dx = \frac{ml}{\pi^2} \int_0^{\frac{\pi x}{l} = \pi} \left(\frac{\pi x}{l}\right) \sin\left(\frac{\pi x}{l}\right) d\left(\frac{\pi x}{l}\right)$$

$$= \frac{ml}{\pi^2} \left[\sin \frac{\pi x}{l} - \left(\frac{\pi x}{l}\right) \cos \frac{\pi x}{l} \right]_0^{\frac{\pi x}{l} = \pi} = \frac{ml}{\pi}$$

$$m_{22} = m \int_0^l \sin^2 \frac{\pi x}{l} dx = m \int_0^l \frac{1}{2} \left[1 - \cos \frac{2\pi x}{l} \right] dx = \frac{ml}{2}$$

11-16 Contd.

$$EI \int_0^l \phi_1'' \phi_1'' dx = 0 \quad EI \int_0^l \phi_1'' \phi_2'' dx = 0$$

$$EI \int_0^l \phi_2'' \phi_2'' dx = EI \left(\frac{\pi}{l}\right)^4 \int_0^l \sin^2 \frac{\pi x}{l} dx = \left(\frac{\pi}{l}\right)^4 EI \frac{l}{2}$$

$$u = C_1 \phi_1(x) + C_2 \phi_2(x)$$

$$U = \frac{1}{2} \int_0^l EI (u'')^2 dx + \frac{1}{2} k_0 u^2(l) \quad \text{where } u^2(l) = C_1^2$$

$$= \frac{1}{2} k_0 C_1^2 + \frac{1}{2} \left(\frac{\pi}{l}\right)^4 EI \frac{l}{2} C_2^2$$

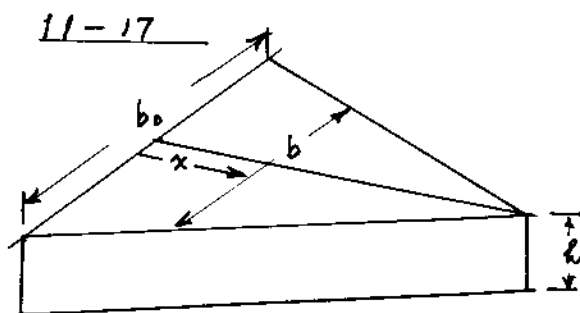
$$\frac{\partial U}{\partial C_1} = k_0 C_1$$

$$\frac{\partial U}{\partial C_2} = \left(\frac{\pi}{l}\right)^4 EI \frac{l}{2} C_2$$

\therefore Eq. 11.3-7 becomes

$$\begin{bmatrix} \left(k_0 - \omega^2 \frac{ml}{3}\right) & -\omega^2 \frac{ml}{\pi} \\ -\omega^2 \frac{ml}{\pi} & \left\{\left(\frac{\pi}{l}\right)^4 EI \frac{l}{2} - \omega^2 \frac{ml}{2}\right\} \end{bmatrix} \begin{Bmatrix} C_1 \\ C_2 \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \end{Bmatrix}$$

$$\left[\left(\frac{l}{6} - \frac{1}{\pi^2}\right)(ml)^2\right] \omega^4 - ml \left[\frac{\pi^4 EI}{6 l^3} + \frac{k_0}{2}\right] \omega^2 + \frac{k_0 \pi^4 EI}{2 l^3} = 0$$



$$m(x) = m_0 \left(1 - \frac{x}{l}\right)$$

$$b(x) = b_0 \left(1 - \frac{x}{l}\right)$$

$$I(x) = \frac{b h^3}{12} = \frac{h^3}{12} b_0 \left(1 - \frac{x}{l}\right)$$

$$\therefore EI = EI_0 \left(1 - \frac{x}{l}\right)$$

$$y = C_1 x^2 + C_2 x^3 = C_1 \phi_1 + C_2 \phi_2$$

$$\phi_1 = x^2$$

$$\phi_2 = x^3$$

$$k_{ij} = \int_0^l EI \phi_i'' \phi_j'' dx$$

$$\phi_1'' = 2$$

$$\phi_2'' = 6x$$

$$m_{ij} = \int_0^l m(x) \phi_i \phi_j dx$$

$$k_{11} = 4EI_0 \int_0^l \left(1 - \frac{x}{l}\right) dx = 4EI_0 \left(l - \frac{l}{2}\right) = 2EI_0 l$$

$$k_{12} = 12EI_0 \int_0^l x \left(1 - \frac{x}{l}\right) dx = 12EI_0 \left(\frac{l^2}{2} - \frac{l^3}{3l}\right) = 2EI_0 l^2$$

$$k_{22} = 36EI_0 \int_0^l x^2 \left(1 - \frac{x}{l}\right) dx = 36EI_0 \left(\frac{l^3}{3} - \frac{l^4}{4l}\right) = 3EI_0 l^3$$

$$m_{11} = m_0 \int_0^l x^4 \left(1 - \frac{x}{l}\right) dx = m_0 \left(\frac{l^5}{5} - \frac{l^6}{6l}\right) = \frac{1}{30} m_0 l^5$$

$$m_{12} = m_0 \int_0^l x^5 \left(1 - \frac{x}{l}\right) dx = m_0 \left(\frac{l^6}{6} - \frac{l^7}{7l}\right) = \frac{1}{42} m_0 l^6$$

$$m_{22} = m_0 \int_0^l x^6 \left(1 - \frac{x}{l}\right) dx = m_0 \left(\frac{l^7}{7} - \frac{l^8}{8l}\right) = \frac{1}{56} m_0 l^7$$

11-17 Cont.

Subst. into Eq. 9.3-7

$$\begin{bmatrix} \left(2EI_0 l - \omega^2 \frac{m_0 l^5}{30}\right) & \left(2EI_0 l^2 - \omega^2 \frac{m_0 l^6}{42}\right) \\ \left(2EI_0 l^2 - \omega^2 \frac{m_0 l^6}{42}\right) & \left(3EI_0 l^3 - \omega^2 \frac{m_0 l^7}{56}\right) \end{bmatrix} \begin{Bmatrix} C_1 \\ C_2 \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \end{Bmatrix}$$

Freq. Eq. from determinant of above eq = 0

$$6(EI_0)^2 l^4 - \omega^2 \left[\frac{m_0 l^5}{30} 3EI_0 l^3 + \frac{m_0 l^7}{56} 2EI_0 l \right] + \omega^4 \left(\frac{m_0 l^5}{30} \cdot \frac{m_0 l^7}{56} \right) - 4(EI_0)^2 l^4 + \left[4EI_0 l^2 \frac{m_0 l^6}{42} \right] \omega^2 - \omega^4 \left(\frac{m_0^2 l^{12}}{42^2} \right) = 0$$

$$\omega^4 \left[m_0^2 l^{12} \left(\frac{1}{30 \times 56} - \frac{1}{42^2} \right) \right] - \omega^2 \left[m_0 EI_0 l^8 \left(\frac{3}{30} + \frac{2}{56} - \frac{4}{42} \right) \right] + 2(EI_0)^2 l^4 = 0$$

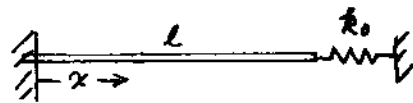
$$(28.345 \times 10^{-6} m_0^2 l^{12}) \omega^4 - (0.0405 m_0 EI_0 l^3) \omega^2 + 2(EI_0)^2 l^4 = 0$$

$$\omega^4 - 1429.3 \left(\frac{EI_0}{m_0 l^4} \right) \omega^2 + 70559 \left(\frac{EI_0}{m_0 l^4} \right)^2 = 0$$

$$\omega^2 = \left\{ \begin{array}{c} 51.20 \\ 1378 \end{array} \right\} \times \frac{EI_0}{m_0 l^4}$$

$$\omega_1 = 7.155 \sqrt{\frac{EI_0}{m_0 l^4}}$$

$$\omega_2 = 37.12 \sqrt{\frac{EI_0}{m_0 l^4}}$$



Normal modes of fixed-free uniform rods are

$$\phi_1 = \sin \frac{\pi}{2} \frac{x}{l}$$

$$\phi_2 = \sin \frac{3\pi}{2} \frac{x}{l} \quad \text{See Eq. 8.2-8}$$

$$\therefore u(x) = C_1 \sin \frac{\pi}{2} \frac{x}{l} + C_2 \sin \frac{3\pi}{2} \frac{x}{l}$$

$$\frac{\partial u}{\partial x} = C_1 \frac{\pi}{2l} \cos \frac{\pi}{2} \frac{x}{l} + C_2 \frac{3\pi}{2l} \cos \frac{3\pi}{2} \frac{x}{l}$$

$$U = \frac{1}{2} AE \int_0^l \left(\frac{\partial u}{\partial x} \right)^2 dx + \frac{1}{2} k_0 u^2(l)$$

$$= \frac{1}{2} AE \int_0^l C_1^2 \left(\frac{\pi}{2l} \right)^2 \cos^2 \frac{\pi}{2} \frac{x}{l} dx + AE C_1 C_2 \int_0^l \left(\frac{\pi}{2l} \right) \left(\frac{3\pi}{2l} \right) \cos \frac{\pi}{2} \frac{x}{l} \cos \frac{3\pi}{2} \frac{x}{l} dx$$

$$+ \frac{1}{2} AE \int_0^l C_2^2 \left(\frac{3\pi}{2l} \right)^2 \cos^2 \frac{3\pi}{2} \frac{x}{l} dx + \frac{1}{2} k_0 (C_1 - C_2)^2$$

$$\frac{\partial U}{\partial C_1} = \left[AE \left(\frac{\pi}{2l} \right)^2 \int_0^l \cos^2 \frac{\pi}{2} \frac{x}{l} dx + k_0 \right] C_1 - k_0 C_2$$

$$+ AE C_2 \int_0^l \left(\frac{\pi}{2l} \right) \left(\frac{3\pi}{2l} \right) \cos \frac{\pi}{2} \frac{x}{l} \cos \frac{3\pi}{2} \frac{x}{l} dx \rightarrow 0$$

$$\frac{\partial U}{\partial C_2} = \left[AE \left(\frac{3\pi}{2l} \right)^2 \int_0^l \cos^2 \frac{3\pi}{2} \frac{x}{l} dx \right] C_2 - k_0 C_1 + k_0 C_2$$

$$\therefore k_{11} = AE \left(\frac{\pi}{2l} \right)^2 \int_0^l \cos^2 \frac{\pi}{2} \frac{x}{l} dx + k_0$$

$$k_{12} = k_{21} = -k_0$$

$$k_{22} = AE \left(\frac{3\pi}{2l} \right)^2 \int_0^l \cos^2 \frac{3\pi}{2} \frac{x}{l} dx + k_0$$

11-18 Cont.

$$\text{Since } \int_0^l \cos^2 n\theta d\theta = \frac{l}{2} \quad n = \frac{\pi}{2l}, \frac{3\pi}{2l},$$

$$k_{11} = AE\left(\frac{\pi}{2l}\right)^2 \frac{l}{2} + k_0$$

$$k_{12} = -k_0$$

$$k_{22} = AE\left(\frac{3\pi}{2l}\right)^2 \frac{l}{2} + k_0$$

$$m_{ij} = \int_0^l m \phi_i \phi_j dx = \begin{cases} m_{11} = \frac{ml}{2} \\ m_{12} = 0 \\ m_{22} = \frac{ml}{2} \end{cases}$$

Subst into Eq. 11.3-7

$$\begin{bmatrix} \left\{ AE\left(\frac{\pi}{2l}\right)^2 \frac{l}{2} + k_0 - \omega^2 \frac{ml}{2} \right\} & -k_0 \\ -k_0 & \left\{ AE\left(\frac{3\pi}{2l}\right)^2 \frac{l}{2} + k_0 - \omega^2 \frac{ml}{2} \right\} \end{bmatrix} \begin{Bmatrix} C_1 \\ C_2 \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \end{Bmatrix}$$

Freq. Eq.

$$\begin{aligned} \left(\frac{ml}{2} \omega^2\right)^2 - \left(\frac{ml}{2} \omega^2\right) \left[AE\left(\frac{\pi}{2l}\right)^2 \frac{l}{2} + k_0 + AE\left(\frac{3\pi}{2l}\right)^2 \frac{l}{2} + k_0 \right] \\ + \left[AE\left(\frac{\pi}{2l}\right)^2 \frac{l}{2} + k_0 \right] \left[AE\left(\frac{3\pi}{2l}\right)^2 \frac{l}{2} + k_0 \right] - k_0^2 = 0 \end{aligned}$$

Reduces to

$$\omega^4 - \omega^2 \left[10 \left(\frac{\pi}{2l}\right)^2 \frac{AE}{m} + \frac{4k_0^2}{ml} \right] + \left[9 \left(\frac{\pi}{2l}\right)^4 \left(\frac{AE}{m}\right)^2 + 20 \left(\frac{\pi}{2l}\right)^2 \frac{AE}{m} \frac{k_0}{ml} \right] = 0$$

11-19

Refer to Prob 11-18

With $k_0 = 0$

$$k_{11} = AE \left(\frac{\pi}{2l} \right)^2 \frac{l}{2} \quad k_{12} = 0$$

$$k_{22} = AE \left(\frac{3\pi}{2l} \right)^2 \frac{l}{2}$$

$$\text{Additional KE} = \frac{1}{2} m_0 \dot{u}(l)^2 = \frac{1}{2} m_0 (\dot{C}_1 - \dot{C}_2)^2$$

$$= \frac{1}{2} m_0 (\dot{C}_1^2 - 2\dot{C}_1\dot{C}_2 + \dot{C}_2^2)$$

$$\frac{d}{dt} \frac{\partial T}{\partial \dot{C}_1} = m_0 \ddot{C}_1 - m_0 \ddot{C}_2$$

$$\frac{d}{dt} \left(\frac{\partial T}{\partial \dot{C}_2} \right) = -m_0 \ddot{C}_1 + m_0 \ddot{C}_2$$

$$\therefore m_{11} = \frac{m_l}{2} + m_0$$

$$m_{12} = m_{21} = -m_0$$

$$m_{22} = \frac{m_l}{2} + m_0$$

Eq. 11.3-7 becomes

$$\begin{bmatrix} \left\{ AE \left(\frac{\pi}{2l} \right)^2 \frac{l}{2} - \omega^2 \left(\frac{m_l}{2} + m_0 \right) \right\} & \omega^2 m_0 \\ \omega^2 m_0 & \left\{ AE \left(\frac{3\pi}{2l} \right)^2 \frac{l}{2} - \omega^2 \left(\frac{m_l}{2} + m_0 \right) \right\} \end{bmatrix} \begin{Bmatrix} C_1 \\ C_2 \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \end{Bmatrix}$$

$$m(x) = m_0 \frac{x}{l} \left(1 - \frac{x}{l}\right)$$

$$\phi_1 = \sin \frac{\pi x}{l}$$

$$\phi_2 = \sin \frac{2\pi x}{l}$$

$$y = \phi_1 q_1 + \phi_2 q_2$$

$$k_{11} = EI \int_0^l \phi_1'' \phi_1'' dx = EI \left(\frac{\pi}{l}\right)^4 \int_0^l \sin^2 \frac{\pi x}{l} dx = EI \left(\frac{\pi}{l}\right)^4 \frac{l}{2}$$

$$k_{12} = k_{21} = EI \int_0^l \phi_1'' \phi_2'' dx = 0$$

$$k_{22} = EI \int_0^l \phi_2'' \phi_2'' dx = EI \left(\frac{2\pi}{l}\right)^4 \int_0^l \sin^2 \frac{2\pi x}{l} dx = EI \left(\frac{2\pi}{l}\right)^4 \frac{l}{2}$$

$$m_{11} = \int_0^l m(x) \phi_1 \phi_1 dx = m_0 \int_0^l \frac{x}{l} \left(1 - \frac{x}{l}\right) \sin^2 \frac{\pi x}{l} dx$$

$$= m_0 \int_0^l \frac{x}{l} \frac{1}{2} \left(1 - \cos \frac{2\pi x}{l}\right) dx = m_0 \int_0^l \left(\frac{x}{l}\right)^2 \frac{1}{2} \left(1 - \cos \frac{2\pi x}{l}\right) dx$$

$$= m_0 \left[\frac{l}{4} - \frac{l}{2\pi^2} \left(\frac{\cos \frac{2\pi x}{l}}{\left(\frac{2\pi}{l}\right)^2} - \left(\frac{\pi x}{l}\right) \frac{\sin \left(\frac{2\pi x}{l}\right)}{\left(\frac{2\pi}{l}\right)} \right) \right]_0^l$$

$$+ m_0 \left[\frac{l}{6} - \frac{l}{16\pi^3} \int_0^l \left(\frac{2\pi x}{l}\right)^2 \cos \frac{2\pi x}{l} d\left(\frac{2\pi x}{l}\right) \right]$$

$$= \frac{m_0 l}{12} + \frac{m_0 l}{16\pi^3} \left[2 \left(\frac{2\pi x}{l}\right) \cos \frac{2\pi x}{l} + \left\{ \left(\frac{2\pi x}{l}\right)^2 - 2 \right\} \sin \frac{2\pi x}{l} \right]_0^l$$

$$= \frac{m_0 l}{12} + \frac{m_0 l}{4\pi^2} = 0.10866 m_0 l$$

11-20 Cont:

$$\begin{aligned}
 m_{22} &= m_0 \int_0^l \left(\frac{x}{l}\right) \left(1 - \frac{x}{l}\right) \sin^2 \frac{2\pi x}{l} dx = m_0 \int_0^l \frac{x}{l} \left(1 - \frac{x}{l}\right) \frac{1}{2} (1 - \cos \frac{4\pi x}{l}) dx \\
 &= \frac{m_0 l}{12} - \frac{m_0}{2} \int_0^l \frac{x}{l} \left(1 - \frac{x}{l}\right) \cos \frac{4\pi x}{l} dx = \frac{m_0 l}{12} + \frac{m_0 l}{(4\pi)^2} \\
 &= 0.08966 m_0 l
 \end{aligned}$$

$$m_{12} = m_0 \int_0^l \left(\frac{x}{l}\right) \left(1 - \frac{x}{l}\right) \sin \frac{\pi x}{l} \sin \frac{2\pi x}{l} dx = 0 \text{ by inspection}$$

ie $\frac{x}{l} \left(1 - \frac{x}{l}\right) = \text{symmetric function}$

$\sin \frac{\pi x}{l} \sin \frac{2\pi x}{l} = \text{unsymmetric function}$

Eq. (1.3-7) becomes

$$\begin{bmatrix} \left[EI \left(\frac{\pi}{l}\right)^4 \frac{l}{2} - 0.10866 m_0 l \omega^2 \right] & [0] \\ [0] & \left[EI \left(\frac{2\pi}{l}\right)^4 \frac{l}{2} - 0.08966 m_0 l \omega^2 \right] \end{bmatrix} \begin{Bmatrix} C_1 \\ C_2 \end{Bmatrix} = 0$$

$\therefore C_1$ & C_2 are independent & Rayleigh-Ritz method fails.

However from above

$$\left[EI \left(\frac{\pi}{l}\right)^4 \frac{l}{2} - 0.10866 m_0 l \omega^2 \right] = 0 \quad \left[EI \left(\frac{2\pi}{l}\right)^4 \frac{l}{2} - 0.08966 m_0 l \omega^2 \right] = 0$$

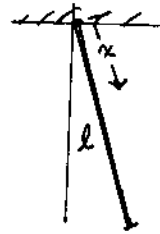
reduces to two Rayleigh's method.

$$\omega_1^2 = \frac{\pi^4}{2 \times 0.10866} \left(\frac{EI}{m_0 l^4} \right) = 448 \frac{EI}{m_0 l^4} \quad \omega_1 = 21.2 \sqrt{\frac{EI}{m_0 l^4}}$$

$$\omega_2^2 = \frac{8\pi^4}{0.08966} \left(\frac{EI}{m_0 l^4} \right) = 8691 \frac{EI}{m_0 l^4} \quad \omega_2 = 93 \sqrt{\frac{EI}{m_0 l^4}}$$

1st mode is $\sin \frac{\pi x}{l}$, 2nd mode is $\sin \frac{2\pi x}{l}$, Compare with Prob 11-11.

11-21



$$\phi_1 = \frac{x}{l} \quad \phi_1'' = 0$$

$$\phi_2 = \sin \frac{\pi x}{l} \quad \phi_2'' = -\left(\frac{\pi}{l}\right)^2 \sin \frac{\pi x}{l}$$

$$\phi_3 = \sin \frac{2\pi x}{l} \quad \phi_3'' = -\left(\frac{2\pi}{l}\right)^2 \sin \frac{2\pi x}{l}$$

$$k_{11} = EI \int_0^l \phi_1'' \phi_1'' dx = 0 \quad k_{12} = k_{13} = 0$$

$$k_{22} = EI \left(\frac{\pi}{l}\right)^4 \int_0^l \sin^2 \frac{\pi x}{l} dx = EI \left(\frac{\pi}{l}\right)^4 \frac{l}{2}$$

$$k_{33} = EI \left(\frac{2\pi}{l}\right)^4 \frac{l}{2}$$

$$k_{23} = EI \left(\frac{\pi}{l}\right)^2 \left(\frac{2\pi}{l}\right)^2 \int_0^l \sin \frac{\pi x}{l} \cdot \sin \frac{2\pi x}{l} dx = 0$$

$$m_{11} = m_0 \int_0^l \phi_1^2 dx = m_0 \int_0^l \left(\frac{x}{l}\right)^2 dx = \frac{m_0 l}{3}$$

$$m_{22} = m_0 \int_0^l \sin^2 \frac{\pi x}{l} dx = \frac{m_0 l}{2}$$

$$m_{33} = m_0 \int_0^l \sin^2 \frac{2\pi x}{l} dx = \frac{m_0 l}{2}$$

$$m_{12} = m_0 \int_0^l \frac{x}{l} \sin \frac{\pi x}{l} dx = \frac{m_0}{l} \left[\frac{\sin \frac{\pi x}{l}}{\left(\frac{\pi}{l}\right)^2} - x \frac{\cos \frac{\pi x}{l}}{\left(\frac{\pi}{l}\right)} \right]_0^l$$

$$= \frac{m_0 l}{\pi} = m_{21}$$

$$m_{13} = m_0 \int_0^l \frac{x}{l} \sin \frac{2\pi x}{l} dx = - \frac{m_0 l}{2\pi}$$

11-21 Cont

$$m_{23} = m_0 \int_0^l \sin \frac{\pi x}{l} \sin \frac{2\pi x}{l} dx = 0$$

$$\begin{bmatrix} (0 - \omega^2 \frac{m_0 l}{3}) & (0 - \omega^2 \frac{m_0 l}{\pi}) & (0 + \omega^2 \frac{m_0 l}{2\pi}) \\ (0 - \omega^2 \frac{m_0 l}{\pi}) & (EI(\frac{\pi}{l})^4 \frac{l}{2} - \omega^2 \frac{m_0 l}{2}) & (0 - 0) \\ (0 - \omega^2 \frac{m_0 l}{2\pi}) & (0 - 0) & (EI(\frac{2\pi}{l})^4 \frac{l}{2} - \omega^2 \frac{m_0 l}{2}) \end{bmatrix} \begin{Bmatrix} C_1 \\ C_2 \\ C_3 \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \\ 0 \end{Bmatrix}$$

Freq. Eq.

$$-\omega^2 \frac{m_0 l}{3} [EI(\frac{\pi}{l})^4 \frac{l}{2} - \omega^2 \frac{m_0 l}{2}] [EI(\frac{2\pi}{l})^4 \frac{l}{2} - \omega^2 \frac{m_0 l}{2}]$$

$$+ \omega^2 \frac{m_0 l}{\pi} \left[-(\omega^2 \frac{m_0 l}{\pi}) \left\{ EI(\frac{2\pi}{l})^4 \frac{l}{2} - \omega^2 \frac{m_0 l}{2} \right\} \right]$$

$$- \omega^2 \frac{m_0 l}{2\pi} \left[-(\omega^2 \frac{m_0 l}{2\pi}) \left\{ EI(\frac{\pi}{l})^4 \frac{l}{2} - \omega^2 \frac{m_0 l}{2} \right\} \right] = 0$$

$$-(\omega^2 m_0 l)^3 \left[\frac{1}{12} - \frac{1}{2\pi^2} + \frac{1}{8\pi^2} \right] + (\omega^2 m_0 l)^2 EI \left[\frac{l}{12} \left(\frac{\pi}{l} \right)^4 + \frac{l}{12} \left(\frac{2\pi}{l} \right)^4 - \frac{l}{2\pi^2} \left(\frac{2\pi}{l} \right)^4 + \frac{l}{8\pi^2} \left(\frac{\pi}{l} \right)^4 \right] - (\omega^2 m_0 l) (EI)^2 \frac{l^2}{12} \left(\frac{\pi}{l} \right)^4 \left(\frac{2\pi}{l} \right)^4 = 0$$

$$-0.045338(\omega^2)^3 + 60.273(\omega^2)^2 \left(\frac{EI}{m_0 l^4} \right) - 12651.37 \omega^2 \left(\frac{EI}{m_0 l^4} \right)^2 = 0$$

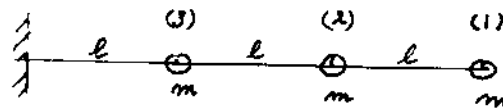
$$(\omega^2)^2 - (1329.4146 \frac{EI}{m_0 l^4}) \omega^2 + 279045.61 \left(\frac{EI}{m_0 l^4} \right)^2 = 0$$

$$\omega^2 = (664.70 \pm 403.47) \left(\frac{EI}{m_0 l^4} \right) = \begin{Bmatrix} 261.23 \\ 1068.17 \end{Bmatrix} \frac{EI}{m_0 l^4}$$

$$\omega = \begin{Bmatrix} 0.0 \\ 16.16 \\ 32.68 \end{Bmatrix} \sqrt{\frac{EI}{m_0 l^4}}$$

$$\text{Exact Sol } \omega = \begin{Bmatrix} 0 \\ 15.4 \\ 50. \end{Bmatrix} \sqrt{\frac{EI}{m_0 l^4}}$$

11-22



From Ex. 6.1-2

$$[a] = \frac{l^3}{3EI} \begin{bmatrix} 27 & 14 & 4 \\ 14 & 8 & 2.5 \\ 4 & 2.5 & 1 \end{bmatrix}$$

$$\begin{Bmatrix} x_1 \\ x_2 \\ x_3 \end{Bmatrix} = \lambda \begin{bmatrix} 27 & 14 & 4 \\ 14 & 8 & 2.5 \\ 4 & 2.5 & 1 \end{bmatrix} \begin{Bmatrix} x_1 \\ x_2 \\ x_3 \end{Bmatrix} \quad \text{where } \lambda = \frac{m\omega^2 l^3}{3EI}$$

Start iteration with

$$\begin{Bmatrix} 1 \\ .3 \\ .1 \end{Bmatrix} \rightarrow \begin{Bmatrix} 31.60 \\ 16.65 \\ .75 \end{Bmatrix} = 31.6 \begin{Bmatrix} 1.00 \\ .5269 \\ .0237 \end{Bmatrix} \rightarrow \begin{Bmatrix} 34.47 \\ 18.27 \\ 5.409 \end{Bmatrix} = 34.47 \begin{Bmatrix} 1.00 \\ .530 \\ .1569 \end{Bmatrix}$$

$$\rightarrow \begin{Bmatrix} 35.047 \\ 18.632 \\ 5.4819 \end{Bmatrix} = 35.047 \begin{Bmatrix} 1.00 \\ .5316 \\ .1564 \end{Bmatrix} \rightarrow \begin{Bmatrix} 35.068 \\ 18.6438 \\ 5.4854 \end{Bmatrix} = 35.068 \begin{Bmatrix} 1.00 \\ .5316 \\ .1564 \end{Bmatrix}$$

$$\therefore 35.068 \frac{m\omega_1^2 l^3}{3EI} = 1$$

$$\omega_1^2 = .0855 \frac{EI}{ml^3}$$

$$\omega_1 = 0.2925 \sqrt{\frac{EI}{ml^3}}$$

$$\begin{Bmatrix} x_1 \\ x_2 \\ x_3 \end{Bmatrix}^{(1)} = \begin{Bmatrix} 1.00 \\ .5316 \\ .1564 \end{Bmatrix}$$

11-22 Cont

2nd mode

$$[S] = \begin{bmatrix} 0 & -\frac{x_2}{x_1} & -\frac{x_3}{x_1} \\ 0 & 1 & 0 \\ 0 & 0 & 1 \end{bmatrix} = \begin{bmatrix} 0 & -.5316 & -.1564 \\ 0 & 1 & 0 \\ 0 & 0 & 1 \end{bmatrix}$$

$$\begin{bmatrix} 27 & 14 & 4 \\ 14 & 8 & 2.5 \\ 4 & 2.5 & 1 \end{bmatrix} \begin{bmatrix} 0 & -.5316 & -.1564 \\ 0 & 1 & 0 \\ 0 & 0 & 1 \end{bmatrix} = \begin{bmatrix} 0 & -.3532 & -.2228 \\ 0 & .5576 & .3184 \\ 0 & .3736 & .3744 \end{bmatrix}$$

Start iteration with

$$\begin{Bmatrix} 1 \\ 0 \\ -.5 \end{Bmatrix} \rightarrow \begin{Bmatrix} .1114 \\ -.1592 \\ -.1868 \end{Bmatrix} = \begin{Bmatrix} 1.00 \\ -1.429 \\ -1.677 \end{Bmatrix} \xrightarrow{2^{nd} \text{ iter}} \begin{Bmatrix} .8784 \\ -1.3308 \\ -1.1617 \end{Bmatrix} = \begin{Bmatrix} 1.00 \\ -1.5150 \\ -1.3225 \end{Bmatrix}$$

$$\xrightarrow{3^{rd} \text{ iter}} \begin{Bmatrix} .8298 \\ -1.2658 \\ -1.0611 \end{Bmatrix} = \begin{Bmatrix} 1.00 \\ -1.5254 \\ -1.2787 \end{Bmatrix} \rightarrow \begin{Bmatrix} .8237 \\ -1.2577 \\ -1.0486 \end{Bmatrix} = \begin{Bmatrix} 1.00 \\ -1.5269 \\ -1.2731 \end{Bmatrix}$$

$$\rightarrow \begin{Bmatrix} .8229 \\ -1.2568 \\ -1.0471 \end{Bmatrix} = \begin{Bmatrix} 1.00 \\ -1.5273 \\ -1.2725 \end{Bmatrix} \rightarrow \begin{Bmatrix} .8229 \\ -1.2568 \\ -1.0470 \end{Bmatrix} = .8229 \begin{Bmatrix} 1.00 \\ -1.5273 \\ -1.2723 \end{Bmatrix}$$

$$\therefore \omega_2^2 = \frac{3}{.8229} \frac{EI}{ml^3} = 3.6456 \left(\frac{EI}{ml^3} \right)$$

$$\omega_2 = 1.9094 \sqrt{\frac{EI}{ml^3}}$$

$$\begin{Bmatrix} x_1 \\ x_2 \\ x_3 \end{Bmatrix}^{(2)} = \begin{Bmatrix} 1.00 \\ -1.5273 \\ -1.2723 \end{Bmatrix}$$

11-22 Cont.

3rd mode

$$C_1 = \bar{x}_1 + .5316 \bar{x}_2 + .1564 \bar{x}_3 = 0$$

$$C_2 = \bar{x}_1 - 1.5273 \bar{x}_2 - 1.2723 \bar{x}_3 = 0$$

Subtr. $\bar{x}_2 = -1.1207 \bar{x}_3$ subst. back into C_1

to get $\bar{x}_1 = .4394 \bar{x}_3$ $\therefore \bar{x}_3 = 2.2758 \bar{x}_1$
 $\bar{x}_2 = -2.5505 \bar{x}_1$

$$[S] = \begin{bmatrix} 0 & 0 & 1.00 \\ 0 & 0 & -2.5505 \\ 0 & 0 & 2.2758 \end{bmatrix} \quad \text{Mult. by } [a]$$

$$\begin{Bmatrix} x_1 \\ x_2 \\ x_3 \end{Bmatrix} = \lambda \begin{bmatrix} 0 & 0 & .3962 \\ 0 & 0 & -.7145 \\ 0 & 0 & -.1005 \end{bmatrix} \begin{Bmatrix} x_1 \\ x_2 \\ x_3 \end{Bmatrix}$$

Start iteration with

$$\begin{Bmatrix} 1 \\ 1 \\ 1 \end{Bmatrix} \rightarrow \begin{Bmatrix} .3962 \\ -.7145 \\ -.1005 \end{Bmatrix} = \begin{Bmatrix} 1.00 \\ -1.8034 \\ -.2537 \end{Bmatrix} \xrightarrow{2^{\text{nd}} \text{ iter.}} \begin{Bmatrix} .1005 \\ -.1812 \\ .0255 \end{Bmatrix} = .1005 \begin{Bmatrix} 1.00 \\ -1.8034 \\ -.2537 \end{Bmatrix}$$

$$\therefore .1005 \frac{m \omega^2 l^3}{3 EI} = 1$$

$$\omega_3^2 = 29.851 \frac{EI}{m l^3}$$

$$\omega_3 = 5.4636 \sqrt{\frac{EI}{m l^3}}$$

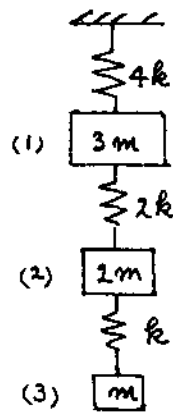
$$\begin{Bmatrix} x_1 \\ x_2 \\ x_3 \end{Bmatrix}^{(3)} = \begin{Bmatrix} 1.00 \\ -1.803 \\ -.2537 \end{Bmatrix}$$

11-23

$$a_{11} = \frac{1}{4k} = a_{21} = a_{12} = a_{31} = a_{13}$$

$$a_{22} = \frac{4k+2k}{4k \cdot 2k} = \frac{3}{4k} = a_{32} = a_{23}$$

$$a_{33} = \frac{1}{4k} + \frac{1}{2k} + \frac{1}{k} = \frac{7}{4k}$$



$$\begin{Bmatrix} x_1 \\ x_2 \\ x_3 \end{Bmatrix} = \frac{m\omega^2}{4k} \begin{bmatrix} 1 & 1 & 1 \\ 1 & 3 & 3 \\ 1 & 3 & 7 \end{bmatrix} \begin{bmatrix} 3 & 0 & 0 \\ 0 & 2 & 0 \\ 0 & 0 & 1 \end{bmatrix} \begin{Bmatrix} x_1 \\ x_2 \\ x_3 \end{Bmatrix} = \lambda [a][m]\{x\}$$

$$= \lambda \begin{bmatrix} 3 & 2 & 1 \\ 3 & 6 & 3 \\ 3 & 6 & 7 \end{bmatrix} \begin{Bmatrix} x_1 \\ x_2 \\ x_3 \end{Bmatrix} \quad \text{Start iteration with } \begin{Bmatrix} 1 \\ 3 \\ 7 \end{Bmatrix}$$

$$\begin{Bmatrix} 1 \\ 3 \\ 7 \end{Bmatrix} \rightarrow \begin{Bmatrix} 16 \\ 48 \\ 70 \end{Bmatrix} = \begin{Bmatrix} 1 \\ 3.0 \\ 4.375 \end{Bmatrix} = \begin{Bmatrix} .229 \\ .686 \\ 1.00 \end{Bmatrix} \quad \text{normalized to } x_3 = 1.0$$

2nd iter.

$$\rightarrow \begin{Bmatrix} 3.059 \\ 7.803 \\ 11.803 \end{Bmatrix} = \begin{Bmatrix} .259 \\ .661 \\ 1.00 \end{Bmatrix} \xrightarrow{3^{\text{rd}} \text{ iter}} \begin{Bmatrix} 3.099 \\ 7.743 \\ 11.743 \end{Bmatrix} = \begin{Bmatrix} .264 \\ .659 \\ 1.00 \end{Bmatrix} \rightarrow$$

$$4^{\text{th}} \rightarrow \begin{Bmatrix} 3.110 \\ 7.746 \\ 11.746 \end{Bmatrix} = \begin{Bmatrix} .265 \\ .660 \\ 1.00 \end{Bmatrix} \xrightarrow{5^{\text{th}}} \begin{Bmatrix} 3.113 \\ 7.749 \\ 11.749 \end{Bmatrix} = \begin{Bmatrix} .265 \\ .660 \\ 1.00 \end{Bmatrix} \times 11.749$$

$$11.749 \frac{m\omega^2}{4k} = 1.0 \quad \omega_1^2 = 0.341 \frac{k}{m}$$

$$\omega_1 = 0.584 \sqrt{\frac{k}{m}}$$

11-23 Cont

2nd mode

$$[S] = \begin{bmatrix} 0 & -\left(\frac{2}{3}\right)\left(\frac{.660}{.265}\right) - \left(\frac{1}{3}\right)\left(\frac{1.00}{.265}\right) \\ 0 & 1 & 0 \\ 0 & 0 & 1 \end{bmatrix} = \begin{bmatrix} 0 & -1.660 & -1.258 \\ 0 & 1 & 0 \\ 0 & 0 & 1 \end{bmatrix}$$

$$\{x\} = \lambda [a][m][S]\{x\}$$

$$\begin{Bmatrix} x_1 \\ x_2 \\ x_3 \end{Bmatrix} = \lambda \begin{bmatrix} 0 & -2.980 & -2.774 \\ 0 & 1.020 & -.774 \\ 0 & 1.020 & 3.226 \end{bmatrix} \begin{Bmatrix} x_1 \\ x_2 \\ x_3 \end{Bmatrix} \quad \text{Start with } \begin{Bmatrix} -.2 \\ .6 \\ 1.0 \end{Bmatrix}$$

$$\rightarrow \begin{Bmatrix} -4.562 \\ -.162 \\ 3.838 \end{Bmatrix} = \begin{Bmatrix} -1.189 \\ -.042 \\ 1.00 \end{Bmatrix} \xrightarrow{2^{nd}} \begin{Bmatrix} -2.649 \\ -.817 \\ 3.183 \end{Bmatrix} = \begin{Bmatrix} -.832 \\ -.257 \\ 1.00 \end{Bmatrix} \xrightarrow{3^{rd}}$$

took 13 iteration to stabilize

$$\xrightarrow{13^{th}} \begin{Bmatrix} -1.466 \\ -1.222 \\ 2.778 \end{Bmatrix} = 2.778 \begin{Bmatrix} -.528 \\ -.440 \\ 1.00 \end{Bmatrix}$$

$$\omega_2^2 = 1.440 \frac{k}{m}$$

$$\omega_2 = 1.200 \sqrt{\frac{k}{m}}$$

— 3rd mode —

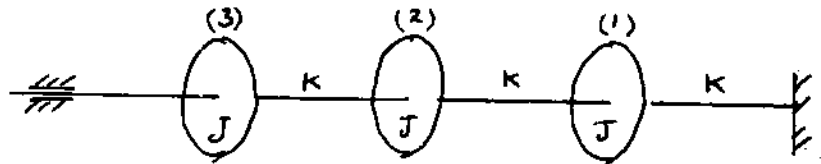
$$[S] = \begin{bmatrix} 0 & 0 & 1.581 \\ 0 & 0 & -1.710 \\ 0 & 0 & 1.00 \end{bmatrix} \quad \begin{Bmatrix} x_1 \\ x_2 \\ x_3 \end{Bmatrix} = \lambda \begin{bmatrix} 0 & 0 & 2.323 \\ 0 & 0 & -2.517 \\ 0 & 0 & 1.483 \end{bmatrix} \begin{Bmatrix} x_1 \\ x_2 \\ x_3 \end{Bmatrix}$$

$$1.483 \frac{m\omega^2}{4k} = 1$$

$$\omega_3^2 = 2.697 \frac{k}{m}$$

$$\omega_3 = 1.642 \sqrt{\frac{k}{m}}$$

11-24



$$a_{11} = a_{21} = a_{12} = a_{31} = a_{13} = \frac{1}{K}$$

$$a_{22} = a_{32} = a_{23} = \frac{1}{K} + \frac{1}{K} = \frac{2}{K}$$

$$a_{33} = \frac{3}{K}$$

$$[a] = \frac{1}{K} \begin{bmatrix} 1 & 1 & 1 \\ 1 & 2 & 2 \\ 1 & 2 & 3 \end{bmatrix}$$

$$\begin{Bmatrix} \theta_1 \\ \theta_2 \\ \theta_3 \end{Bmatrix} = \frac{\omega^2 J}{K} \begin{bmatrix} 1 & 1 & 1 \\ 1 & 2 & 2 \\ 1 & 2 & 3 \end{bmatrix} \begin{Bmatrix} \theta_1 \\ \theta_2 \\ \theta_3 \end{Bmatrix}$$

Start with

$$\begin{Bmatrix} 1 \\ 1 \\ 1 \end{Bmatrix} \rightarrow \begin{Bmatrix} 3 \\ 5 \\ 6 \end{Bmatrix} = \begin{Bmatrix} .50 \\ .833 \\ 1.00 \end{Bmatrix} \rightarrow \begin{Bmatrix} 2.337 \\ 4.166 \\ 5.166 \end{Bmatrix} = \begin{Bmatrix} .4576 \\ .8065 \\ 1.000 \end{Bmatrix} \rightarrow \begin{Bmatrix} 2.2581 \\ 4.0645 \\ 5.0646 \end{Bmatrix} = \begin{Bmatrix} .4458 \\ .8015 \\ 1.000 \end{Bmatrix}$$

$$\rightarrow \begin{Bmatrix} 2.2483 \\ 4.0508 \\ 5.0508 \end{Bmatrix} = \begin{Bmatrix} .4451 \\ .8020 \\ 1.000 \end{Bmatrix} \rightarrow \begin{Bmatrix} 2.2471 \\ 4.0491 \\ 5.0491 \end{Bmatrix} = \begin{Bmatrix} .4450 \\ .8019 \\ 1.000 \end{Bmatrix} \rightarrow \begin{Bmatrix} 2.2469 \\ 4.0488 \\ 5.0488 \end{Bmatrix} = \begin{Bmatrix} .4450 \\ .8019 \\ 1.000 \end{Bmatrix}$$

$$\omega_1 = \sqrt{\frac{1}{5.0488} \frac{K}{J}} = 0.4450 \sqrt{\frac{K}{J}}$$

2nd mode

$$\bar{\theta}_1 = -\frac{.8019}{.4450} \bar{\theta}_2 - \frac{1.00}{.4450} \bar{\theta}_3 = -1.8020 \bar{\theta}_2 - 2.2472 \bar{\theta}_3$$

$$[S] = \begin{bmatrix} 0 & -1.8020 & -2.2472 \\ 0 & 1 & 0 \\ 0 & 0 & 1 \end{bmatrix}$$

11-24 Cont.

$$\begin{Bmatrix} \theta_1 \\ \theta_2 \\ \theta_3 \end{Bmatrix} = \frac{\omega^2 J}{K} \begin{bmatrix} 0 & -.8020 & -1.2472 \\ 0 & .1980 & -.2472 \\ 0 & .1980 & .7528 \end{bmatrix} \begin{Bmatrix} \theta_1 \\ \theta_2 \\ \theta_3 \end{Bmatrix} \text{ converges to } \begin{Bmatrix} -.8019 \\ -.3571 \\ .6429 \end{Bmatrix} = \begin{Bmatrix} -1.2473 \\ -.5339 \\ 1.000 \end{Bmatrix}$$

$$\omega_2 = \sqrt{\frac{1}{.6429} \frac{K}{J}} = 1.247 \sqrt{\frac{K}{J}}$$

3rd mode

$$C_1 = 0 \text{ gives } \bar{\theta}_1 = -1.802 \bar{\theta}_2 - 2.2472 \bar{\theta}_3$$

$$C_2 = 0 \text{ " } \bar{\theta}_1 = -.4451 \bar{\theta}_2 + .8018 \bar{\theta}_3$$

$$0 = -1.3569 \bar{\theta}_2 - 3.0490 \bar{\theta}_3$$

$$\bar{\theta}_2 = -2.247 \bar{\theta}_3$$

$$\bar{\theta}_1 = 1.8020 \bar{\theta}_3$$

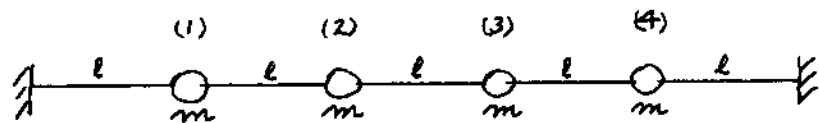
$$[S]_2 = \begin{bmatrix} 0 & 0 & 1.8020 \\ 0 & 0 & -2.247 \\ 0 & 0 & 1.00 \end{bmatrix}$$

$$\begin{Bmatrix} \theta_1 \\ \theta_2 \\ \theta_3 \end{Bmatrix} = \frac{\omega^2 J}{K} \begin{bmatrix} 0 & -.8020 & -1.2472 \\ 0 & .1980 & -.2472 \\ 0 & .1980 & .7528 \end{bmatrix} \begin{bmatrix} 0 & 0 & 1.8020 \\ 0 & 0 & -2.247 \\ 0 & 0 & 1.00 \end{bmatrix}$$

$$= \frac{\omega^2 J}{K} \begin{bmatrix} 0 & 0 & .5557 \\ 0 & 0 & -.6921 \\ 0 & 0 & .3079 \end{bmatrix} \begin{Bmatrix} \theta_1 \\ \theta_2 \\ \theta_3 \end{Bmatrix} \text{ converges to } \begin{Bmatrix} .5557 \\ .6921 \\ .3079 \end{Bmatrix} = \begin{Bmatrix} 1.803 \\ -2.248 \\ 1.000 \end{Bmatrix}$$

$$\omega_3 = \sqrt{\frac{1}{.3079} \frac{K}{J}} = 1.802 \sqrt{\frac{K}{J}}$$

11-25



$T = \text{tension}$

From unit defl. at each pt. $\sum F$

$$a_{11} = \frac{4}{5} \frac{l}{T}$$

$$a_{21} = a_{12} = \frac{3}{5} \frac{l}{T}$$

$$a_{31} = a_{13} = \frac{2}{5} \frac{l}{T}$$

$$a_{41} = \frac{1}{5} \frac{l}{T}$$

$$a_{22} = \frac{6}{5} \frac{l}{T}$$

$$a_{32} = \frac{4}{5} \frac{l}{T}$$

$$a_{42} = \frac{2}{5} \frac{l}{T}$$

$$a = \frac{l}{5T} \begin{bmatrix} 4 & 3 & 2 & 1 \\ 3 & 6 & 4 & 2 \\ 2 & 4 & 6 & 3 \\ 1 & 2 & 3 & 4 \end{bmatrix}$$

$$m = m \begin{bmatrix} 1 & & & \\ & 1 & & \\ & & 1 & \\ & & & 1 \end{bmatrix}$$

$$\begin{Bmatrix} y_1 \\ y_2 \\ y_3 \\ y_4 \end{Bmatrix} = \frac{\omega^2 m l}{5T} \begin{bmatrix} 4 & 3 & 2 & 1 \\ 3 & 6 & 4 & 2 \\ 2 & 4 & 6 & 3 \\ 1 & 2 & 3 & 4 \end{bmatrix} \begin{Bmatrix} y_1 \\ y_2 \\ y_3 \\ y_4 \end{Bmatrix} \quad \text{start with } \begin{Bmatrix} .5 \\ 1.0 \\ 1.0 \\ .5 \end{Bmatrix}$$

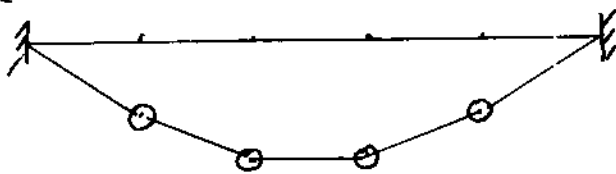
$$\rightarrow \begin{Bmatrix} 7.5 \\ 12.50 \\ 12.50 \\ 7.5 \end{Bmatrix} = \begin{Bmatrix} 1.00 \\ 1.666 \\ 1.666 \\ 1.00 \end{Bmatrix} \rightarrow \begin{Bmatrix} 13.333 \\ 21.666 \\ 21.666 \\ 13.333 \end{Bmatrix} = \begin{Bmatrix} 1.000 \\ 1.625 \\ 1.625 \\ 1.000 \end{Bmatrix} \rightarrow \text{etc}$$

Converges to

$$\begin{Bmatrix} 13.095 \\ 21.190 \\ 21.190 \\ 13.095 \end{Bmatrix} = 13.095 \begin{Bmatrix} 1.000 \\ 1.618 \\ 1.618 \\ 1.000 \end{Bmatrix}$$

$$\omega_1^2 = \frac{5}{13.09} \frac{T}{m l}$$

$$\omega_1 = 0.618 \sqrt{\frac{T}{m l}}$$



Symmetric
0 - nodes

11-25 Cont:

2nd mode

$$[S]_1 = \begin{bmatrix} 0 & -1.618 & -1.618 & -1.000 \\ 0 & 1 & 0 & 0 \\ 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 1 \end{bmatrix}$$

$$Y = \lambda [a][S]_1 Y, \quad \lambda = \frac{\omega^2 m l}{5 T}$$

$$\begin{Bmatrix} y_1 \\ y_2 \\ y_3 \\ y_4 \end{Bmatrix} = \lambda \begin{bmatrix} 0 & -3.4720 & -4.4720 & -3.000 \\ 0 & 1.1460 & -0.8540 & -1.000 \\ 0 & 0.7640 & 2.7640 & 1.000 \\ 0 & 0.3820 & 1.3820 & 3.000 \end{bmatrix} \begin{Bmatrix} y_1 \\ y_2 \\ y_3 \\ y_4 \end{Bmatrix}$$

2nd mode will be unsymmetric \therefore start with

$$\begin{Bmatrix} 1.0 \\ -6 \\ -6 \\ -1.0 \end{Bmatrix} \rightarrow \begin{Bmatrix} 3.6000 \\ 2.2000 \\ -2.2000 \\ -3.6000 \end{Bmatrix} = \begin{Bmatrix} -1.000 \\ -0.6111 \\ 0.6111 \\ 1.000 \end{Bmatrix} \rightarrow \begin{Bmatrix} -3.6111 \\ -2.2222 \\ 2.2222 \\ 3.6111 \end{Bmatrix} = \begin{Bmatrix} -1.000 \\ -0.6154 \\ 0.6154 \\ 1.000 \end{Bmatrix}$$

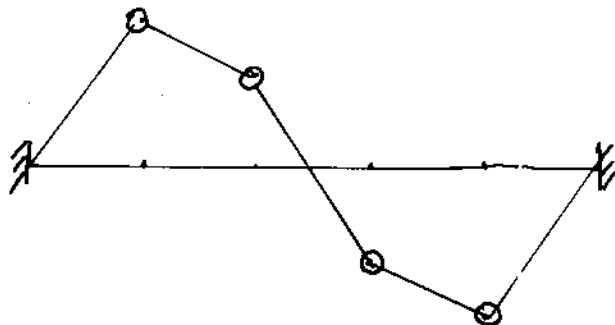
etc. converges after 4 more iterations to

$$\begin{Bmatrix} -3.6180 \\ -2.2360 \\ 2.2360 \\ 3.3180 \end{Bmatrix} = 3.3180 \begin{Bmatrix} -1.000 \\ -0.6180 \\ 0.6180 \\ 1.000 \end{Bmatrix}$$

$$\omega_2^2 = \frac{5}{3.318} \frac{T}{m l}$$

$$= 1.3820 \frac{T}{m l}$$

$$\omega_2 = 1.1756 \sqrt{\frac{T}{m l}}$$



antisymmetric

1 node

11-25 Cont:

3rd mode

$$C_1 = 0 = 1.0 \bar{y}_1 + 1.618 \bar{y}_2 + 1.618 \bar{y}_3 + 1.00 \bar{y}_4$$

$$C_2 = 0 = -1.0 \bar{y}_1 - .618 \bar{y}_2 + .618 \bar{y}_3 + 1.00 \bar{y}_4$$

adding $1.00 \bar{y}_2 + 2.236 \bar{y}_3 + 2.00 \bar{y}_4 = 0$

$$\bar{y}_2 = -2.236 \bar{y}_3 - 2.00 \bar{y}_4$$

$$[S]_2 = \begin{bmatrix} 0 & -.618 & .618 & 1.00 \\ 0 & 0 & -2.236 & -2.0 \\ 0 & 0 & 1.0 & 0 \\ 0 & 0 & 0 & 1.0 \end{bmatrix}$$

$$Y = \lambda [a][S][S]_2 Y$$

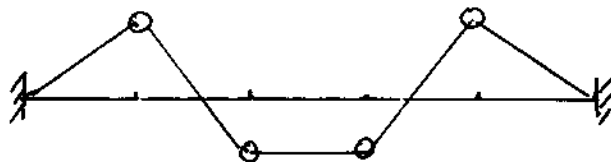
$$\begin{Bmatrix} y_1 \\ y_2 \\ y_3 \\ y_4 \end{Bmatrix} = \lambda \begin{bmatrix} 0 & 0 & 3.2914 & 3.9440 \\ 0 & 0 & -3.4165 & -3.2920 \\ 0 & 0 & 1.0557 & -.5280 \\ 0 & 0 & .5278 & 2.2360 \end{bmatrix} \begin{Bmatrix} y_1 \\ y_2 \\ y_3 \\ y_4 \end{Bmatrix}$$

3rd mode must have 2 modes & be symmetric, start with

$$\begin{Bmatrix} 1.0 \\ -.6 \\ -.6 \\ 1.0 \end{Bmatrix} \rightarrow \text{converges to} \begin{Bmatrix} 1.9098 \\ -1.1806 \\ -1.1806 \\ 1.9098 \end{Bmatrix} = \begin{Bmatrix} 1.0000 \\ -.6182 \\ -.6182 \\ 1.0000 \end{Bmatrix}$$

$$\omega_3^2 = \frac{5}{1.9098} \frac{T}{mL} = 2.618 \frac{T}{mL}$$

$$\omega_3 = 1.618 \sqrt{\frac{T}{mL}}$$



symmetric
2 modes

11-25 Cont: 4th mode

$$C_1 = 0 = 1.0 \bar{y}_1 + 1.618 \bar{y}_2 + 1.618 \bar{y}_3 + 1.00 \bar{y}_4$$

$$C_2 = 0 = -1.00 \bar{y}_1 - .618 \bar{y}_2 + .618 \bar{y}_3 + 1.00 \bar{y}_4$$

$$C_3 = 0 = 1.00 \bar{y}_1 - .618 \bar{y}_2 - .618 \bar{y}_3 + 1.00 \bar{y}_4$$

Solving

$$\bar{y}_2 = 1.618 \bar{y}_4 \quad \bar{y}_3 = -1.618 \bar{y}_4$$

$$\bar{y}_1 = 0.618 \bar{y}_3$$

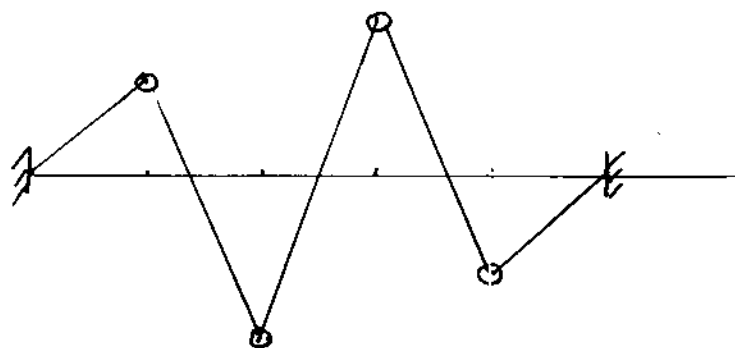
$$[S]_3 = \begin{bmatrix} 0 & 0 & .618 & 0 \\ 0 & 0 & 0 & 1.618 \\ 0 & 0 & 0 & -1.618 \\ 0 & 0 & 0 & 1.0 \end{bmatrix}$$

$$Y = \lambda [am][s]_1 [s]_2 [s]_3 Y$$

$$\begin{Bmatrix} y_1 \\ y_2 \\ y_3 \\ y_4 \end{Bmatrix} = \lambda \begin{bmatrix} 0 & 0 & 0 & -1.3815 \\ 0 & 0 & 0 & 2.2357 \\ 0 & 0 & 0 & -2.2361 \\ 0 & 0 & 0 & 1.3820 \end{bmatrix} \begin{Bmatrix} y_1 \\ y_2 \\ y_3 \\ y_4 \end{Bmatrix}$$

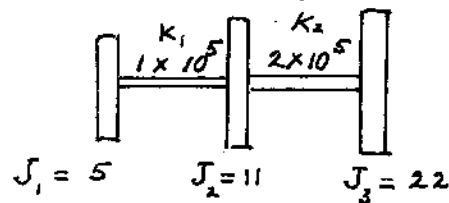
$$\omega_a^2 = \frac{5}{1.382} \frac{T}{ml} = 3.6178 \frac{T}{ml}$$

$$\omega_a = 1.902 \sqrt{\frac{T}{ml}}$$



3 nodes
antisymmetric

12-1 Using the HP-25 programmable calculator the following simple program consisting of 36 program steps is written



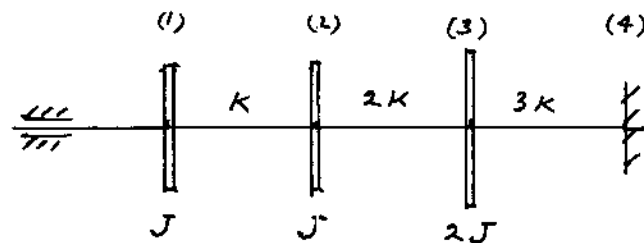
Program Mode	Program Gives	Run Mode	Display
00 [F] [PRGM]		[F] [PRGM]	
02 1 [↑]	θ_1	ω [9] 2	
04 5 [↑]	J_1	[STO] 0	
06 [RCL] 0 [X]	$\omega^2 J_1$	[R/S]	T_3
07 [STO] 1	$\omega^2 J_1$	[RCL] 2	θ_2
12 1 [EEEX] 5 [÷] [—]	$\theta_2 = 1 - \frac{\omega^2 J_1}{K_1}$	[RCL] 4	θ_3
13 [STO] 2	θ_2	[RCL] 0	ω^2
14 11 [X]	$J_2 \theta_2$	[F] [Y]	ω
18 [RCL] 0 [X]	$\omega^2 J_2 \theta_2$		
20 [RCL] 1 [+	$T_2 = \omega^2 J_1 + \omega^2 J_2 \theta_2$		
21 [STO] 3	T_2		
25 2 [EEEX] 5 [÷]	T_2 / K_2		
28 [RCL] 2 [—] [CHS]	$\theta_3 = \theta_2 - \frac{T_2}{K_2}$		
29 [STO] 4	θ_3		
32 22 [X]	$J_3 \theta_3$		
34 [RCL] 0 [X]	$\omega^2 J_3 \theta_3$		
36 [RCL] 3 [+	$T_3 = T_2 + \omega^2 J_3 \theta_3$ Newton meters		

The first column indicates the number of steps (49 are allowed). After the program is keyed in, switch to the Run Mode. Choosing any ω , ω^2 is stored in space 0 and by pressing [R/S] the calculations in the program mode is executed automatically with T_3 displayed. [RCL] 2 and [RCL] 4 then displays θ_2 and θ_3 respectively. To repeat for any other frequency it is only necessary to input the new ω in the run mode and press [R/S].

12-1 Cont. The following ω_s were keyed in to obtain T_3 and θ_2 θ_3 at natural frequencies

ω	$T_3 \times 10^{-3}$	θ_1	θ_2	θ_3
20	14.66	Mode Shape		
40	52.32 ($T_3 = 52.32 \times 10^3 \text{ Nm}$)			
60	95.43			
80	119.38			
100	99.50			
120	20.75			
123.66	-0.0118	1.00	0.2353	-0.3449
140	-109.65			
160	-246.96			
180	-290.74			
200	-64.00			
202	-16.78			
202.6584	.00134	1.00	-1.0535	0.2995
203	8.95			
205	64.89			
220	710.55 Nm			

12-2



$$J = 1 \text{ kg.m}^2$$

$$K = 0.2 \times 10^6 \frac{\text{Nm}}{\text{r.}}$$

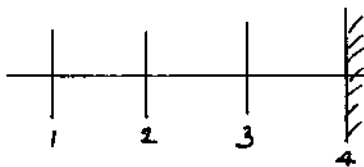
This problem may be solved by digital computer using Eqs. (12.21) & (12.2-2), however for these simple systems, the hand calculator can be easily programmed, as in Prob. 12-1. Only slight changes are necessary.

1st mode was also obtained by matrix iteration as $\omega_1 = 281.3$

$$\text{Eq. for iteration} \rightarrow \begin{Bmatrix} \theta_1 \\ \theta_2 \\ \theta_3 \end{Bmatrix} = \frac{J\omega^2}{6K} \begin{bmatrix} 11 & 5 & 2 \\ 5 & 5 & 2 \\ 2 & 2 & 2 \end{bmatrix} \begin{bmatrix} 1 & 0 & 0 \\ 0 & 1 & 0 \\ 0 & 0 & 2 \end{bmatrix} \begin{Bmatrix} \theta_1 \\ \theta_2 \\ \theta_3 \end{Bmatrix}$$

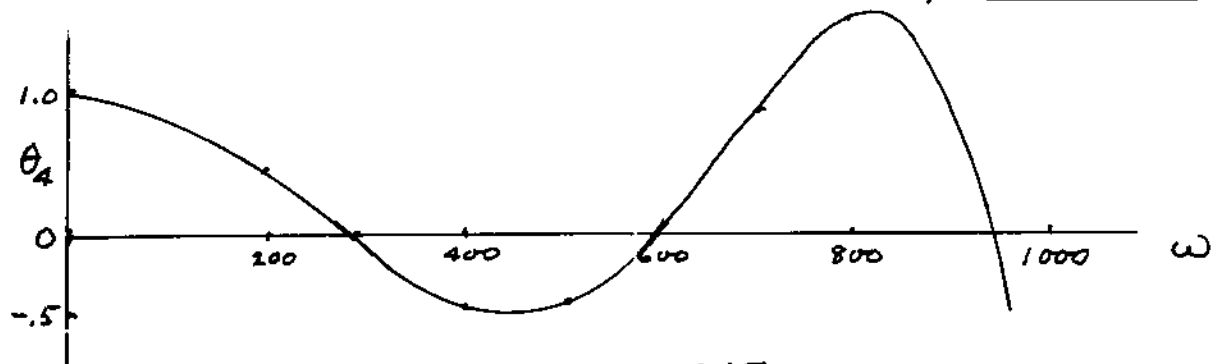
12-2 Cont: Use same computer program HP-25 as in Prob. 1 -1 with following changes.

		<u>Run Mode</u>	<u>Display</u>
00			
02			
04	1 \uparrow	Same	$\boxed{R/S}$ θ_4
06			
07			\boxed{RCL} 2 θ_2
12	2 \boxed{EEX} 5 $\boxed{\div}$ $\boxed{-}$		\boxed{RCL} 4 θ_3
13			
14	1 $\boxed{\times}$		
18			
20			
21			
25	4 \boxed{EEX} 5 $\boxed{\div}$		
28			
29			
32	2 $\boxed{\times}$		
34			
36			
40	6 \boxed{EEX} 5 $\boxed{\div}$		
44	\boxed{RCL} 4 $\boxed{-}$ \boxed{CHS}		



Results:

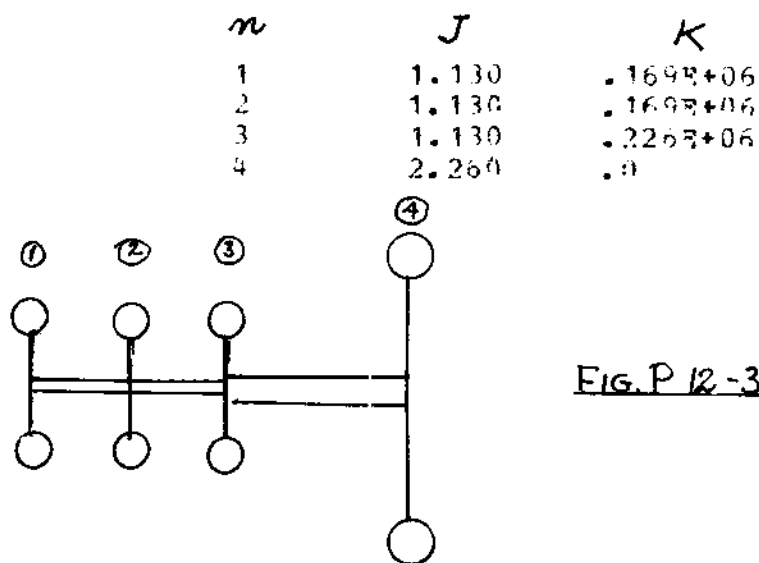
ω	θ_1	θ_2	θ_3	θ_4	
20	1.00	.9980	.9960	.9933	
200				.417	
281.24	1.00	.6045	.2872	-.00000812	<u>1st Mode</u>
400				-.4507	
500				-.432	
589.368	1.00	-.7368	-.9654	-.0000082	<u>2nd Mode</u>
700.				.936	
800				1.597	
900				.8954	
934.6382	1.00	-3.3677	1.8031	.0000067	<u>3rd Mode</u>



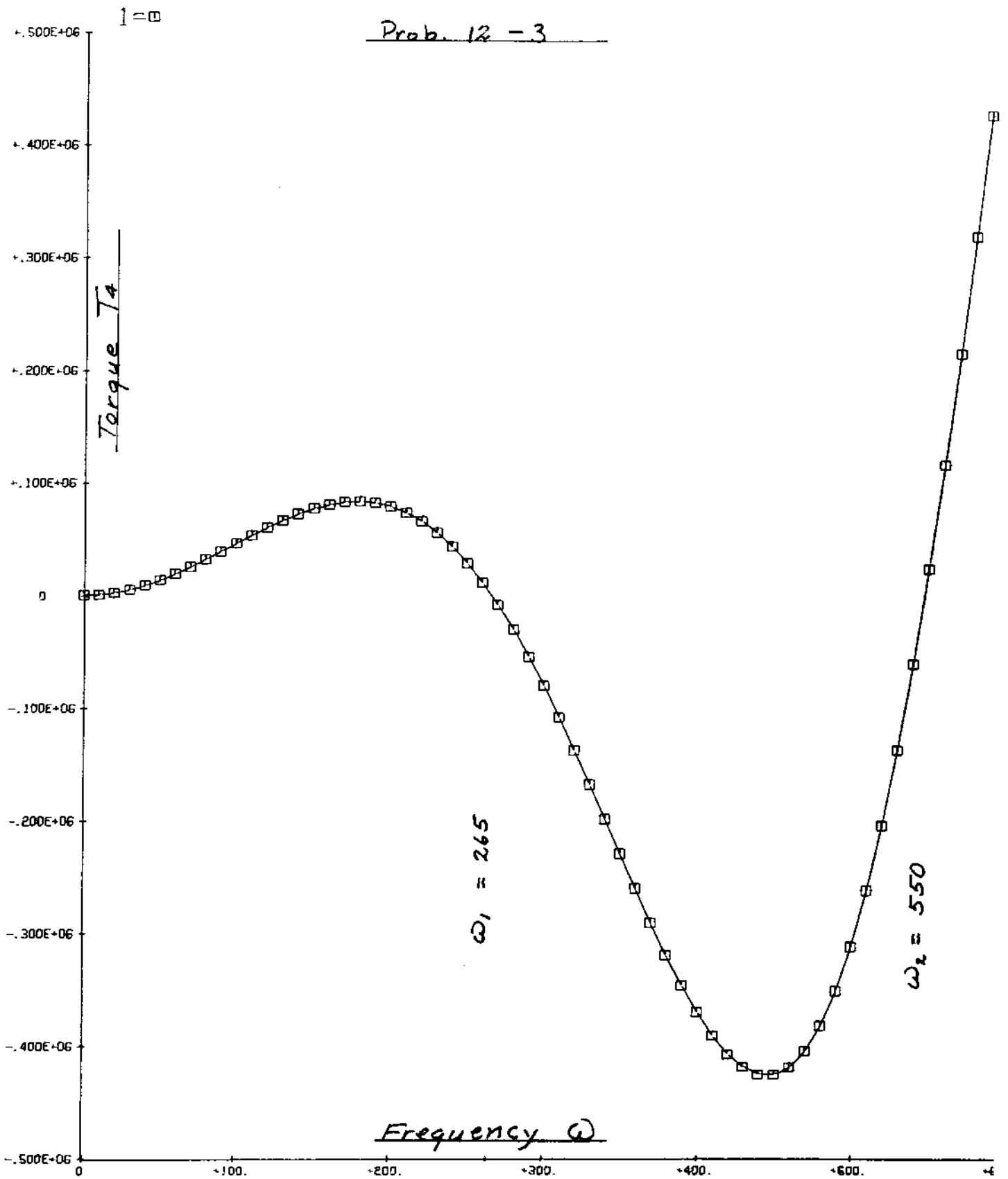
```

C      PROBLEM 10.3 THOMSON
0002  DIMENSION RJ(4),RK(4),V(60),DE(60,4),T(60,4),TF(60)
0003  M=4
0004  L=60
0005  READ5,(RJ(J),J=1,M)
0006  5 FORMAT(4F10.3)
0007  READ6,(RK(J),J=1,M)
0008  6 FORMAT(4E10.3)
0009  DO 20 I=1,L
0010  DE(I,1)=1.
0011  W(I)=(I-1)*10
0012  T(I,1)=W(I)**2*RJ(1)*DE(I,1)
0013  DO 10 J=2,M
0014  DE(I,J)=DE(I,J-1)-T(I,J-1)/RK(J-1)
0015  T(I,J)=T(I,J-1)+W(I)**2*RJ(J)*DE(I,J)
0016  10 CONTINUE
0017  20 CONTINUE
0018  DO 25 J=1,M
0019  PRINT24,J,RJ(J),RK(J)
0020  24 FORMAT(20X,I3,5X,F8.3,5X,E8.3)
0021  25 CONTINUE
0022  DO 40 I=1,L
0023  T2(I)=T(I,M)
0024  PRINT30,W(I),DE(I,M),T(I,M)
0025  30 FORMAT(10X,F8.2,10X,E12.4,10X,E12.4)
0026  40 CONTINUE
0027  CALL EXPLOTT(W,TF,L)
0028  CALL FINISH
0029  STOP
0030  END

```



<u>12-3</u>	<u>ω</u>	<u>θ_4</u>	<u>T_4</u>
	0.0	1.0000	0.0
	10.00	0.9965	563.9053
	20.00	0.9860	2242.5156
	30.00	0.9686	4996.6680
	40.00	0.9444	8761.6445
	50.00	0.9135	13447.9648
	60.00	0.8761	18942.5312
	70.00	0.8325	25110.0469
	80.00	0.7829	31794.7812
	90.00	0.7276	38822.5586
	100.00	0.6670	46003.1094
	110.00	0.6015	53132.5893
	120.00	0.5315	59996.4297
	130.00	0.4574	66372.3125
	140.00	0.3797	72033.5000
	150.00	0.2990	76752.1875
	160.00	0.2157	80303.0000
	170.00	0.1304	82466.8750
	180.00	0.0438	83034.6875
	190.00	-0.0436	81811.2500
	200.00	-0.1312	78618.8750
	210.00	-0.2183	73301.9375
	220.00	-0.3042	65729.9375
	230.00	-0.3884	55801.7617
	240.00	-0.4701	43449.1133
	250.00	-0.5488	28640.1875
	260.00	-0.6236	11382.6250
	270.00	-0.6941	-8273.0625
	280.00	-0.7596	-30231.3125
	290.00	-0.8194	-54348.7500
	300.00	-0.8730	-80432.5000
	310.00	-0.9197	-108238.687
	320.00	-0.9590	-137471.437
	330.00	-0.9905	-167782.687
	340.00	-1.0137	-198772.437
	350.00	-1.0280	-229990.312
	360.00	-1.0332	-260936.125
	370.00	-1.0290	-291063.687
	380.00	-1.0150	-319784.562
	390.00	-0.9911	-346471.562
	400.00	-0.9571	-370465.062
	410.00	-0.9131	-391080.000
	420.00	-0.8591	-407613.687
	430.00	-0.7952	-419355.500
	440.00	-0.7216	-425595.937
	450.00	-0.6387	-425642.500
	460.00	-0.5470	-418828.125
	470.00	-0.4471	-404532.187
	480.00	-0.3395	-382193.375
	490.00	-0.2253	-351330.875
	500.00	-0.1054	-311564.437
	510.00	0.0192	-262637.062
	520.00	0.1470	-204442.125
	530.00	0.2767	-137046.375
	540.00	0.4066	-60724.3125
	550.00	0.5349	24013.4375
	560.00	0.6595	116383.625
	570.00	0.7782	215294.125



Program Mode

00 [f] [PRGM]

02 1 [↑]

04 [RCL] 0 [-]

05 [STO] 1

07 1 [↑]

09 [RCL] 0 [X]

12 [RCL] 1 [-] [CHS]

13 [STO] 2

15 [RCL] 1 [↑]

17 1 [↑]

19 [RCL] 0 [X]

22 [RCL] 2 [-] [CHS]

$x_1 = 1$

$m_1 = 1$

$1 - \omega^2 m_1 = x_2$

x_2

$(1 + x_2)$

$\omega^2(1 + x_2) = F_2$

$x_3 = x_2 - F_2$

x_3

$x_2 + x_3$

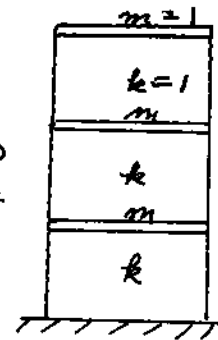
$1 + x_2 + x_3$

$\omega^2(1 + x_2 + x_3) = F_3$

$= x_4$

Run Mode

[f] [PRGM]

 ω [↑] = ω [g] 2 = ω^2 [STO] 0 = ω^2 [R/S] = x_4 [RCL] 2 = x_3 [RCL] 1 = x_2 

$\rightarrow x_1 = 1$

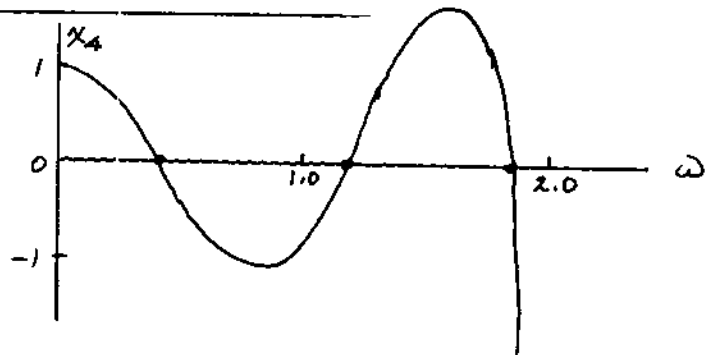
$\rightarrow x_2$

$\rightarrow x_3$

$x_4 = 0$

Freq. Plot. Scan.

ω	x_4
$\omega_1 \rightarrow .2$.768
.6	-.5587
.8	-1.054
1.0	-1.000
$\omega_2 \rightarrow 1.5$	1.422
1.7	1.283
$\omega_3 \rightarrow 1.9$	-2.545



ω	x_4	x_3	x_2	x_1	
$\omega_1 = .44504$.0000068	.4450	.8019	1.00	$\therefore \omega_1 = .44504 \sqrt{\frac{k}{m}}$
$\omega_2 = 1.247$.0001	-1.247	-.535	1.00	$\omega_2 = 1.247 \sqrt{\frac{k}{m}}$
$\omega_3 = 1.802$	-.0012	1.8027	-2.247	1.00	$\omega_3 = 1.802 \sqrt{\frac{k}{m}}$

Mode Shapes

$$12-5$$

$$F_1 = m\omega^2$$

$$x_2 = 1 - \frac{m\omega^2}{k}$$

$$F_2 = 2m\omega^2 x_2$$

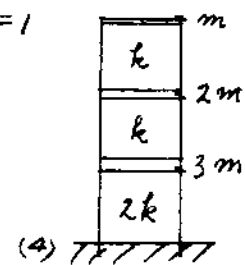
$$F_1 + F_2 = m\omega^2 + 2m\omega^2 x_2 \quad x_3 = x_2 - \frac{F_1 + F_2}{k}$$

$$F_3 = 3m\omega^2 x_3$$

$$F_1 + F_2 + F_3 = m\omega^2 + 2m\omega^2 x_2 + 3m\omega^2 x_3$$

$$x_4 = x_3 - \frac{F_1 + F_2 + F_3}{2k}$$

$$x_1 = 1$$



Prog f PRGM

1 ↑

RCL 0 - STO 1 = x_2

2 X 1 + STO 2

RCL 0 X STO 3

RCL 1 RCL 3 - = x_3

STO 4 = x_3

3 X RCL 2 +

RCL 0 X 2 ÷ CHS

RCL 4 + = x_4

STO 5 = x_4

Run Mode

F PRGM

ω ↑

g 2 = ω^2

STO 0 = ω^2

R/S = x_4

RCL 1 = x_2

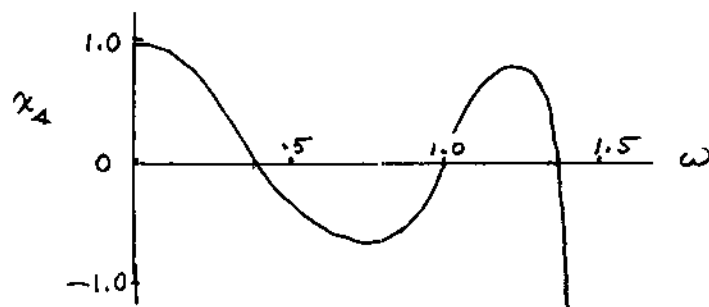
RCL 4 = x_3

Result only of nat. freq.

n	ω_n	x_4	x_3	x_2	x_1
1	.4385	-.0004	.3330	.8164	1.00
2	1.000	.000	-1.00	.000	1.00
3	1.3478	.0003	.3336	-.8166	1.00

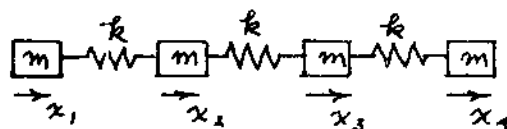
$$\therefore \omega_1 = .4385 \sqrt{\frac{k}{m}}$$

etc.



12-7 Same program as Prob 10-4 but extended to $F_4 = 0$

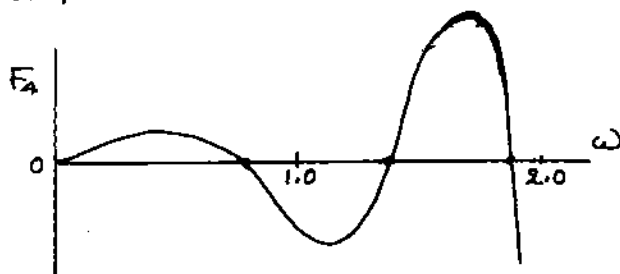
Program identical through 22



```

22  - - - - -
23  STO 3      x4
25  RCL 2 +    x3 + x4
27  RCL 1 +    x2 + x3 + x4
29  1 +        1 + x2 + x3 + x4
31  RCL 0 X    ω²(1 + x2 + x3 + x4) = F4
  
```

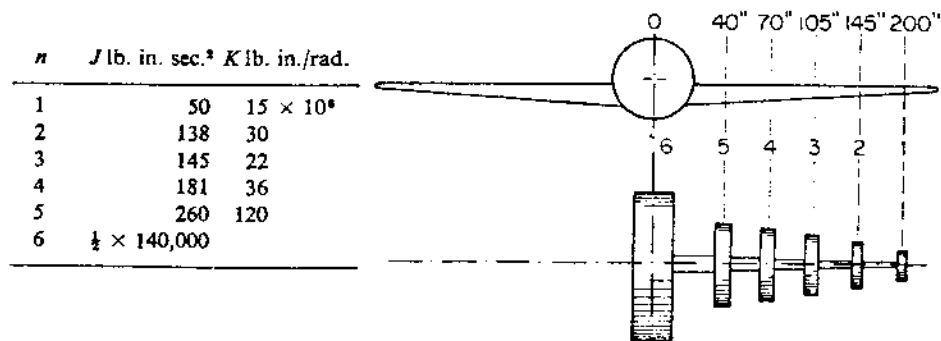
ω	F_4	x_4	x_3	x_2	x_1
			<u>Mode Shape</u>		
.01	.0004				
.2	.144				
.4	.408				
.6	.407				
.76537	-.000011	-1.00	-.4142	.4142	1.00
.8	-.131				
1.0	-1.000				
1.4	-1.567				
1.4142	-.0002	1.00	-1.00	-1.00	1.00
1.6	2.417				
1.8	1.858				
1.848	.0	-1.00	2.418	-2.418	1.00
2.0	-16.00				
2.4	-262.0				



$$\omega_1 = .76537 \sqrt{\frac{k}{m}}$$

$$\omega_2 = 1.4142 \sqrt{\frac{k}{m}}$$

$$\omega_3 = 1.848 \sqrt{\frac{k}{m}}$$



Sta.	J	K
1	50.000	0.150E+08
2	138.000	0.300E+08
3	145.000	0.220E+08
4	181.000	0.360E+08
5	260.000	0.120E+09
6	70000.000	0.0

```

PROBLEM 10.9 THOMSON, ANTISYMMETRIC
DIMENSION RJ(15),RK(15),W(200),DE(200,15),T(200,15),TF(200)
M=6
L=100
READ5,(RJ(J),J=1,M)
5 FORMAT(6F10.3)
READ6,(RK(J),J=1,M)
6 FORMAT(6E10.3)
DO 20 I=1,L
  DE(I,1)=1.
  W(I)=(I-1)*10
  T(I,1)=W(I)**2*RJ(1)*DE(I,1)
  DO 10 J=2,M
    DE(I,J)=DE(I,J-1)-T(I,J-1)/RK(J-1)
    T(I,J)=T(I,J-1)+W(I)**2*RJ(J)*DE(I,J)
10 CONTINUE
20 CONTINUE
DO 25 J=1,M
  PRINT24,J,RJ(J),RK(J)
24 FORMAT(20X,I3,5X,F10.3,5X,E10.3)
25 CONTINUE
DO 40 I=1,L
  TF(I)=T(I,M)
  PRINT30,W(I),DE(I,M),T(I,M)
30 FORMAT(10X,F8.2,10X,F12.4,10X,E12.4)
40 CONTINUE
CALL PZPLOT(W,TF,L)
CALL FINISH
STOP
END

```


ω	Θ_c	T_c
0.0	1.0000	0.0
10.00	0.9955	0.7045E+07
20.00	0.9819	0.2780E+08
30.00	0.9594	0.6113E+08
40.00	0.9283	0.1052E+09
50.00	0.8889	0.1574E+09
60.00	0.8416	0.2147E+09
70.00	0.7870	0.2733E+09
80.00	0.7256	0.3294E+09
90.00	0.6581	0.3784E+09
100.00	0.5852	0.4158E+09
110.00	0.5078	0.4372E+09
120.00	0.4267	0.4381E+09
130.00	0.3428	0.4143E+09
140.00	0.2570	0.3621E+09
150.00	0.1702	0.2781E+09
160.00 ω , Antisym.	0.0834	0.1600E+09
170.00 ω , Symm.	-0.0023	0.5965E+07
180.00	-0.0862	-0.1848E+09
190.00	-0.1672	-0.4122E+09
200.00	-0.2446	-0.6743E+09
210.00	-0.3173	-0.9702E+09
220.00	-0.3846	-0.1295E+10
230.00	-0.4459	-0.1644E+10
240.00	-0.5004	-0.2012E+10
250.00	-0.5475	-0.2392E+10
260.00	-0.5869	-0.2775E+10
270.00	-0.6180	-0.3154E+10
280.00	-0.6406	-0.3519E+10
290.00	-0.6546	-0.3859E+10
300.00	-0.6598	-0.4166E+10
310.00	-0.6564	-0.4427E+10
320.00	-0.6445	-0.4634E+10
330.00	-0.6244	-0.4777E+10
340.00	-0.5966	-0.4848E+10
350.00	-0.5615	-0.4837E+10
360.00	-0.5197	-0.4740E+10
370.00	-0.4721	-0.4552E+10
380.00	-0.4195	-0.4270E+10
390.00	-0.3628	-0.3894E+10
400.00	-0.3030	-0.3426E+10
410.00	-0.2411	-0.2871E+10
420.00	-0.1783	-0.2236E+10
430.00	-0.1157	-0.1532E+10
440.00	-0.0545	-0.7713E+09
450.00	0.0042	0.2731E+08
460.00	0.0592	0.8467E+09
470.00	0.1094	0.1665E+10
480.00	0.1539	0.2457E+10
490.00	0.1916	0.3199E+10
500.00	0.2218	0.3863E+10

12-B Cont.

ω	θ_6	T_6
510.00	0.2436	0.4422E+10
520.00	0.2566	0.4848E+10
530.00	0.2603	0.5115E+10
540.00	0.2546	0.5198E+10
550.00	0.2394	0.5075E+10
560.00	0.2149	0.4729E+10
570.00	0.1817	0.4147E+10
580.00	0.1403	0.3323E+10
590.00	0.0918	0.2258E+10
600.00	0.0372	0.9615E+09
610.00	-0.0219	-0.5465E+09
620.00	-0.0840	-0.2237E+10
630.00	-0.1472	-0.4068E+10
640.00	-0.2095	-0.5989E+10
650.00	-0.2693	-0.7935E+10
660.00	-0.3227	-0.9833E+10
670.00	-0.3690	-0.1160E+11
680.00	-0.4056	-0.1314E+11
690.00	-0.4302	-0.1436E+11
700.00	-0.4411	-0.1516E+11
710.00	-0.4366	-0.1546E+11
720.00	-0.4159	-0.1515E+11
730.00	-0.3784	-0.1419E+11
740.00	-0.3247	-0.1253E+11
750.00	-0.2563	-0.1018E+11
760.00	-0.1759	-0.7202E+10
770.00	-0.0878	-0.3728E+10
780.00	0.0010	0.1259E+08
790.00	0.0849	0.3671E+10
800.00	0.1506	0.6753E+10
810.00	0.1853	0.8583E+10
820.00	0.1722	0.8265E+10
830.00	0.0905	0.4643E+10
840.00	-0.0848	-0.3753E+10
850.00	-0.3835	-0.1876E+11
860.00	-0.8410	-0.4266E+11

Comments: System was cut at centerline using $\frac{1}{2}$ the J of the fuselage. Antisymmetric modes are given for $\theta_6 = 0$ and Symmetric modes for $T_6 = 0$. Note that they are very close to each other because of the very large J_6

12-9

From conservation of momentum

$$M y_0 = 2m y$$

$$\frac{M}{m} = n$$

$$(y + y_0) = \frac{Pl^3}{3EI} = \frac{(m\omega^2 y)l^3}{3EI} = y + \frac{2m}{M} y = y \left(1 + \frac{2}{n}\right)$$

$$\therefore \omega^2 = \frac{3EI}{Ml^3} (n+2) = \frac{6EI}{Ml^3} \left(1 + \frac{n}{2}\right) \quad \omega = \sqrt{\frac{6EI}{Ml^3} \left(1 + \frac{n}{2}\right)}$$

12-10

Using Eq. 12.9-5

$$\begin{Bmatrix} -V_3 \\ M_3 \\ 0 \\ 0 \end{Bmatrix} = \begin{bmatrix} 1 & 0 & 0 & m\omega^2 \\ l & 1 & 0 & m\omega^2 l \\ 3\alpha l & 6\alpha & 1 & m\omega^2 3\alpha l \\ \alpha l^2 & 3\alpha l & l & (1+m\omega^2 \alpha l^2) \end{bmatrix}^2 \begin{Bmatrix} -0 \\ 0 \\ 1 \\ \theta_1 \end{Bmatrix} \quad \text{where} \quad \alpha = \frac{l}{6EI}$$

Calculation can be limited to last 2 rows of last two columns.

$$\left. \begin{aligned} u_{33} + u_{34} \theta_1 &= 0 \\ u_{43} + u_{44} \theta_1 &= 0 \end{aligned} \right\} \text{ or } \frac{u_{33} u_{44} - u_{34} u_{43}}{\text{freq. eq.}} = 0$$

$$\text{For } A^2 = A A = U, \quad u_{ij} = \sum_k a_{ik} a_{kj}$$

$$\therefore u_{33} = \sum_k a_{3k} a_{k3} = 1 + m\omega^2 3\alpha l^2$$

$$\begin{aligned} u_{44} &= \sum_k a_{4k} a_{k4} = m\omega^2 \alpha l^2 + m\omega^2 3\alpha l^2 + m\omega^2 3\alpha l^2 + (1+m\omega^2 \alpha l^2)^2 \\ &= 1 + 9m\omega^2 \alpha l^2 + (m\omega^2 \alpha l^2)^2 \end{aligned}$$

$$\begin{aligned} u_{34} &= \sum_k a_{3k} a_{k4} = m\omega^2 3\alpha l + m\omega^2 6\alpha l + m\omega^2 3\alpha l + m\omega^2 3\alpha l (1+m\omega^2 \alpha l^2) \\ &= 15m\omega^2 \alpha l + 3(m\omega^2 \alpha l)^2 l \end{aligned}$$

$$u_{43} = l + l(1+m\omega^2 \alpha l^2) = 2l + m\omega^2 \alpha l^3$$

12-10 Cont: subst. into freq. eq.

$$(1 + m\omega^2 \alpha l^2) [1 + 9\omega^2 m \alpha l^2 + (m\omega^2 \alpha l^2)^2] \\ + (2l + m\omega^2 \alpha l^3) [-15m\omega^2 \alpha l - 3l(m\omega^2 \alpha l)^2] = 0$$

or $1 - 18(m\omega^2 \alpha l^2) + 7(m\omega^2 \alpha l^2)^2 = 0$

let $\beta = m\omega^2 \alpha l^2$ then $\beta^2 - \frac{18}{7}\beta + \frac{1}{7} = 0$

$$\beta = 1.2857 \pm \sqrt{1.6531 - .1429} = 1.2857 \pm 1.2289$$

$$\beta = \begin{cases} .0568 \\ 2.5615 \end{cases} \quad \sqrt{\beta} = \begin{cases} .2384 \\ 1.5857 \end{cases} \quad \omega = \begin{cases} .584 \\ 3.884 \end{cases} \sqrt{\frac{EI}{ml^3}}$$

12-11

Fortran H program given here can be used for other 3 mass cantilever system.

If all m_s are equal & all l_s are equal, the results for $m=1$, $l=1$, and $EI=1 \times 10^6$ can be used for other values of m and l and EI by evaluating β_1 and β_2 .

Prob. 11-11. Cont.

SYSTEM/370 FORTRAN H EXTENDED (ENHANCED)

```
PROBLEM 10.12 THOMSON
DIMENSION ZV(2,4), ZM(2,4), ZD(2,4), ZY(2,4), WM(3), WL(3), WEI(3),
1 DE(200), DY(200), W(200)
M=4
L=200
N=M-1
READ5, (WM(J), J=1, N)
5 FORMAT(3F10.3)
READ6, (WL(J), J=1, N)
6 FORMAT(3F10.3)
READ7, (WEI(J), J=1, N)
7 FORMAT(3E10.3)
ZD(1,1)=0.0
ZD(2,1)=1.0
ZY(1,1)=1.0
ZY(2,1)=0.0
DO 60 K=1, L
W(K)=(K-1)*10.
DO 50 I=1, 2
ZV(I,1)=0.0
ZM(I,1)=0.0
DO 40 J=2, M
ZV(I,J)=W(K)**2*WM(J-1)*ZY(I,J-1)+ZV(I,J-1)
ZM(I,J)=W(K)**2*WM(J-1)*WL(J-1)*ZY(I,J-1)+ZM(I,J-1)+WL(J-1)*
1 ZV(I,J-1)
ZD(I,J)=W(K)**2*WM(J-1)*WL(J-1)**2/(2.*WEI(J-1))*ZY(I,J-1)+
1 ZD(I,J-1)+WL(J-1)/WEI(J-1)*ZM(I,J-1)+WL(J-1)**2/(2.*WEI(J-1))*
1 ZV(I,J-1)
ZY(I,J)=(1.+W(K)**2*WM(J-1)*WL(J-1)**3/(6.*WEI(J-1)))*ZY(I,J-1)+
1 WL(J-1)*ZD(I,J-1)+WL(J-1)**2/(2.*WEI(J-1))*ZM(I,J-1)+WL(J-1)**3/
1 (6.*WEI(J-1))*ZV(I,J-1)
40 CONTINUE
50 CONTINUE
DE(K)=-ZD(1,M)/ZD(2,M)
DY(K)=ZY(1,M)+ZY(2,M)*DE(K)
60 CONTINUE
DO 70 J=1, N
PRINT65, WM(J), WL(J), WEI(J)
65 FORMAT(10X,F10.3,5X,F10.3,5X,F10.3)
70 CONTINUE
DO 80 K=1, L
PRINT75, W(K), DY(K)
75 FORMAT(20X,F8.2,5X,F12.4)
80 CONTINUE
CALL PZPLOT(W,DY,L)
CALL FINISH
STOP
END
```

12-11 Cont.:

m	l	EI
1.000	1.000	0.100E+07
1.000	1.000	0.100E+07
1.000	1.000	0.100E+07

ω	y_4	ω	y_4
150.00	0.6955	1760.00	-0.4357
160.00	0.6456	1770.00	-0.4084
170.00	0.6038	1780.00	-0.3803
180.00	0.5605	1790.00	-0.3532
190.00	0.5156	1800.00	-0.3255
200.00	0.4694	1810.00	-0.2977
210.00	0.4220	1820.00	-0.2698
220.00	0.3734	1830.00	-0.2418
230.00	0.3239	1840.00	-0.2137
240.00	0.2735	1850.00	-0.1855
250.00	0.2224	1860.00	-0.1572
260.00	0.1707	1870.00	-0.1289
270.00	0.1185	1880.00	-0.1005
280.00	0.0660	1890.00	-0.0720
290.00	0.0131	1900.00	-0.0434
300.00	-0.0398	1910.00	-0.0148
310.00	-0.0920	1920.00	0.0139
320.00	-0.1458	1930.00	0.0427
330.00	-0.1987	1940.00	0.0715
340.00	-0.2513	1950.00	0.1004
350.00	-0.3036	1960.00	0.1293
360.00	-0.3556	1970.00	0.1583
370.00	-0.4071	1980.00	0.1873
380.00	-0.4581	1990.00	0.2164

other data deleted.

Data obtained for $m=1$, $l=1$, and $EI=1 \times 10^6$

For other values of m , l , and EI , first evaluate β from $\omega_i = \beta_i \sqrt{\frac{EI}{ml^3}}$. This

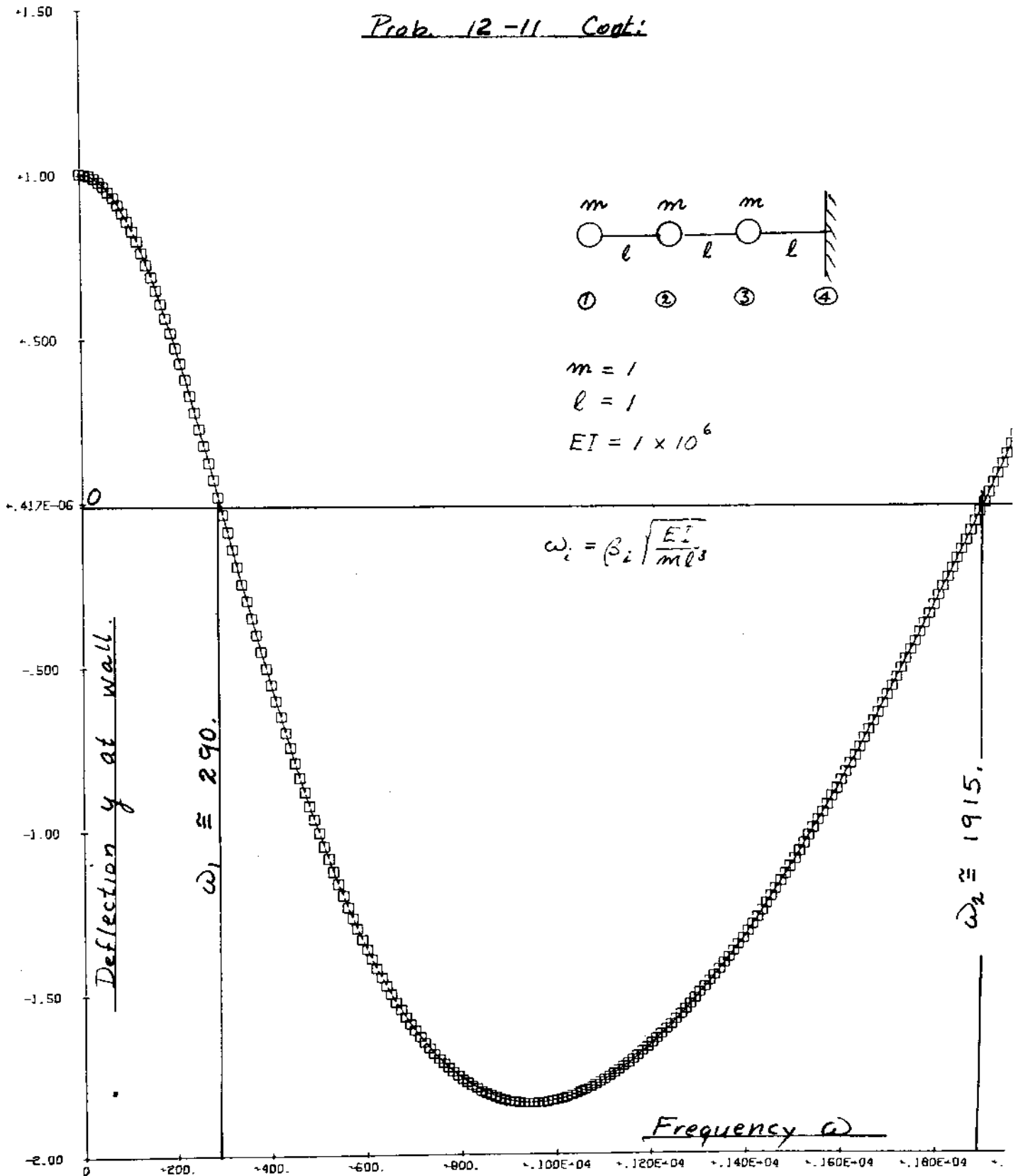
β can then be used for other m , l , & EI .

ie. for 1st mode $\omega_1 = \beta_1 \sqrt{\frac{10^6}{1}}$ $\therefore \beta_1 = \omega_1 \times 10^{-3}$

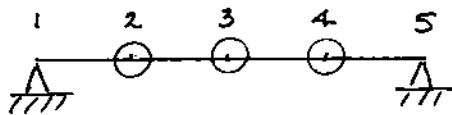
≈ 0.290 From Prob 11-22 $\omega_1 = 0.2925 \sqrt{\frac{EI}{ml^3}}$

$\omega_2 = 1.909 \sqrt{\frac{EI}{ml^3}}$ β_2 from computer sol. = 1.915

Prob. 12-11. Cont:



12-12



stations must be numbered as 1 to 5 with
 $m_1 = m_5 = 0$

$$\begin{Bmatrix} -V \\ 0 \\ \theta \\ 0 \end{Bmatrix}_5 = \begin{bmatrix} - & - & - & - \\ u_{21} & - & u_{23} & - \\ - & - & - & - \\ u_{41} & - & u_{43} & - \end{bmatrix} \begin{Bmatrix} -V \\ 0 \\ \theta \\ 0 \end{Bmatrix}_1$$

$$\therefore \begin{vmatrix} u_{21} & u_{23} \\ u_{41} & u_{43} \end{vmatrix} = 0 \quad \text{or} \quad u_{43} = \frac{u_{41} u_{23}}{u_{21}}$$

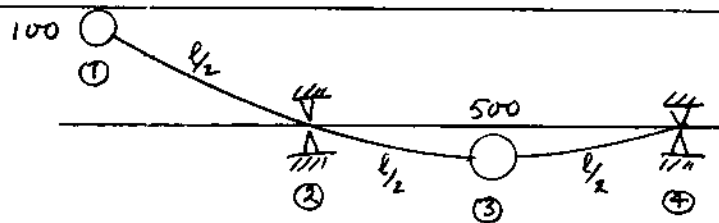
12-13

$$m_2 = m_4 = 0$$

$$y_1 = 1.0$$

$$y_2 = y_4 = 0$$

Eq. 1, 1-5



$$\begin{Bmatrix} -V \\ M \\ \theta \\ 0 \end{Bmatrix}_2 = \begin{bmatrix} \text{sec 1} \end{bmatrix} \begin{Bmatrix} 0 \\ 0 \\ \theta_1 \\ 1 \end{Bmatrix}_1 \quad \begin{Bmatrix} -V \\ M \\ \theta \\ y \end{Bmatrix}_3 = \begin{bmatrix} \text{sec 2} \end{bmatrix} \begin{Bmatrix} -V \\ M \\ \theta \\ 0 \end{Bmatrix}_2 \quad \text{etc}$$

this is constraint eq.

$$\begin{Bmatrix} -V \\ 0 \\ \theta \\ 0 \end{Bmatrix}_4 = \begin{bmatrix} 0 & 0 \\ \text{sec 1} & 0 \\ 0 & 0 \\ 0 & 0 \end{bmatrix} \begin{bmatrix} \text{sec 2} & 0 \\ 0 & 0 \\ 0 & 0 \end{bmatrix} \begin{bmatrix} \text{sec 3} \end{bmatrix} \begin{Bmatrix} 0 \\ 0 \\ \theta_1 \\ 1 \end{Bmatrix}_1 = \begin{bmatrix} - & - & - & - \\ - & - & u_{23} & u_{24} \\ - & - & - & - \\ - & - & u_{43} & u_{44} \end{bmatrix} \begin{Bmatrix} 0 \\ 0 \\ \theta_1 \\ 1 \end{Bmatrix}$$

$$\begin{vmatrix} u_{23} & u_{24} \\ u_{43} & u_{44} \end{vmatrix} = 0$$

12-14

Eqs. 12.4-1 to 6 can be arranged in the following matrix form

$$\begin{Bmatrix} -V \\ M \\ \theta \\ y \\ T \\ \varphi \end{Bmatrix}_{i+1} = \begin{bmatrix} 1 & 0 & 0 & m\omega^2 & 0 & m\omega^2 c \\ l & 1 & 0 & m\omega^2 l & 0 & m\omega^2 cl \\ \frac{l^2}{2EI} & \frac{l}{EI} & 1 & \frac{m\omega^2 l^2}{2EI} & 0 & \frac{m\omega^2 cl^2}{2EI} \\ \frac{l^2}{6EI} & \frac{l^2}{2EI} & l & (1 + \frac{m\omega^2 l^3}{6EI}) & 0 & \frac{m\omega^2 cl^3}{6EI} \\ 0 & 0 & 0 & m\omega^2 c & 1 & J\omega^2 \\ 0 & 0 & 0 & m\omega^2 ch & h & (1 + J\omega^2 h) \end{bmatrix} \begin{Bmatrix} -V \\ M \\ \theta \\ y \\ T \\ \varphi \end{Bmatrix}_i$$

$$\begin{Bmatrix} V \\ W \end{Bmatrix}_3 = \begin{bmatrix} A & 2B \\ 2C & D \end{bmatrix} \begin{bmatrix} A & B \\ C & D \end{bmatrix} \begin{Bmatrix} V \\ W \end{Bmatrix}_1$$

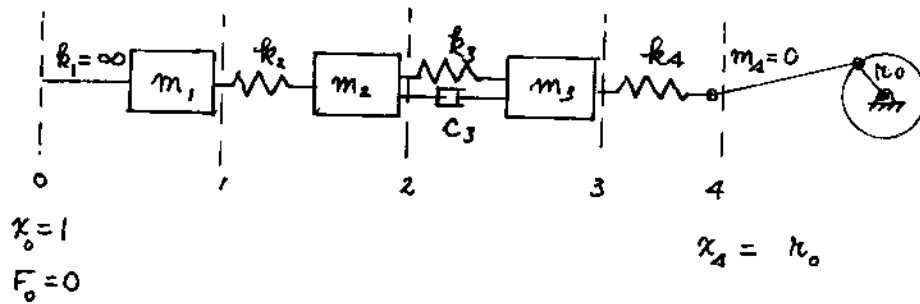
$$= \begin{bmatrix} (A^2 + 2BC) & (AB + 2BD) \\ (2AC + CD) & (2BC + D^2) \end{bmatrix} \begin{Bmatrix} V \\ W \end{Bmatrix}_1$$

Section 2 differs from Sec 1 by $2C_i$ in place of C_i .

$$\begin{Bmatrix} -V_3 \\ M_3 \\ 0 \\ 0 \\ T_3 \\ 0 \end{Bmatrix} = \begin{bmatrix} u_{ij} \end{bmatrix} \begin{Bmatrix} 0 \\ 0 \\ \theta_1 \\ 1 \\ 0 \\ \varphi_1 \end{Bmatrix}$$

$$\therefore \begin{bmatrix} u_{33} & u_{34} & u_{36} \\ u_{43} & u_{44} & u_{46} \\ u_{63} & u_{64} & u_{66} \end{bmatrix} \begin{Bmatrix} \theta_1 \\ 1 \\ \varphi_1 \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \\ 0 \end{Bmatrix}$$

Set determinant to zero for nat. freqs.



$$\begin{Bmatrix} x \\ F \end{Bmatrix}_1 = \begin{Bmatrix} 1 \\ -m_1 \omega^2 \end{Bmatrix} = \begin{bmatrix} 1 & \frac{1}{\infty} \\ -m_1 \omega^2 & 1 - \frac{m_1 \omega^2}{\infty} \end{bmatrix} \begin{Bmatrix} 1 \\ 0 \end{Bmatrix} = \begin{Bmatrix} 1 \\ -m_1 \omega^2 \end{Bmatrix}$$

$$\begin{Bmatrix} x \\ F \end{Bmatrix}_2 = \begin{bmatrix} 1 & \frac{1}{k_2} \\ -m_2 \omega^2 & (1 - \frac{m_2 \omega^2}{k_2}) \end{bmatrix} \begin{Bmatrix} x \\ F \end{Bmatrix}_1$$

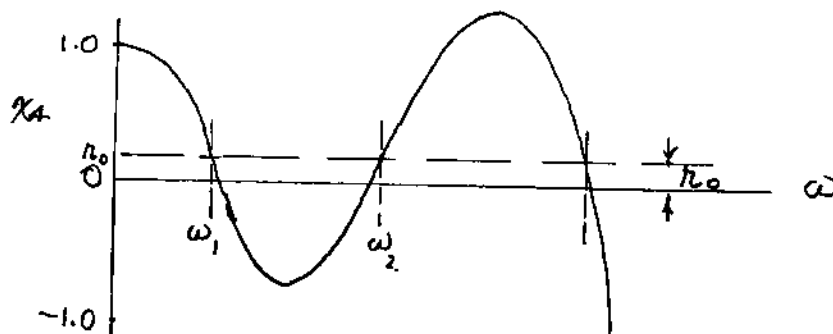
$$\begin{Bmatrix} x \\ F \end{Bmatrix}_3 = \begin{bmatrix} 1 & \frac{1}{k_3 + i\omega c_3} \\ -m_3 \omega^2 & (1 - \frac{m_3 \omega^2}{k_3 + i\omega c_3}) \end{bmatrix} \begin{Bmatrix} x \\ F \end{Bmatrix}_2$$

$$\begin{Bmatrix} x \\ F \end{Bmatrix}_4 = \begin{bmatrix} 1 & \frac{1}{k_4} \\ 0 & 0 \end{bmatrix} \begin{Bmatrix} x \\ F \end{Bmatrix}_3$$

Boundary Cond.

$$\therefore x_4 = x_3 + \frac{F_3}{k_4} = \theta_0$$

Problem is identical in calculation to torsional system and same program as Prob 12-10 can be used.



$$\begin{Bmatrix} \theta \\ T \end{Bmatrix}_0^R = \begin{Bmatrix} 1 \\ -\omega^2 J_1 \end{Bmatrix}$$

$$\begin{Bmatrix} \theta \\ T \end{Bmatrix}_m^R = \begin{bmatrix} 1 & 0 \\ -\omega^2 J & 1 \end{bmatrix}_m \begin{bmatrix} 1 & \frac{1}{k+i\omega g} \\ 0 & 1 \end{bmatrix}_m \begin{Bmatrix} \theta \\ T \end{Bmatrix}_{m-1}^R = \begin{bmatrix} 1 & \frac{1}{k+i\omega g} \\ -\omega^2 J & 1 - \frac{\omega^2 J}{k+i\omega g} \end{bmatrix}_m \begin{Bmatrix} \theta \\ T \end{Bmatrix}_{m-1}^R$$

Elements of above transfer matrix are:

$$\frac{1}{k+i\omega g} = \left[\frac{k}{k^2 + (\omega g)^2} \right] - i \left[\frac{\omega g}{k^2 + (\omega g)^2} \right]$$

$$1 - \frac{\omega^2 J}{k+i\omega g} = \left[1 - \frac{\omega^2 J k}{k^2 + (\omega g)^2} \right] + i \left[\frac{\omega^2 J \omega g}{k^2 + (\omega g)^2} \right]$$

Sample calc. for $\omega^2 = 0.5 \times 10^5$

Renumber J_5 from J_0 to J_3

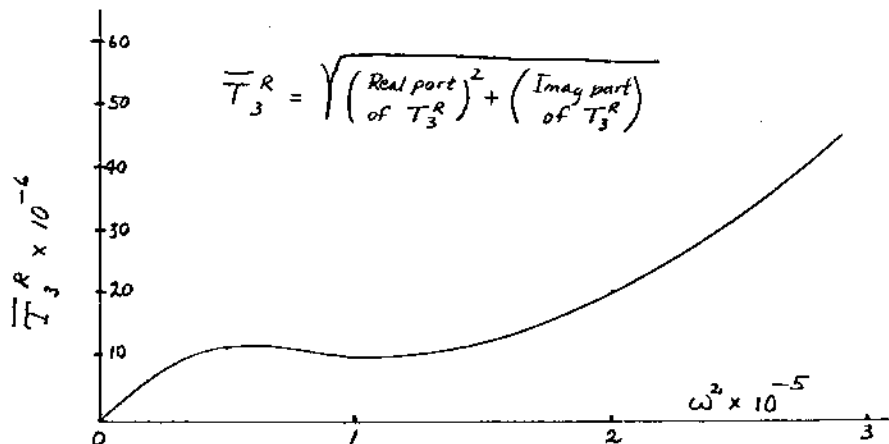
$$\begin{Bmatrix} \theta \\ T \end{Bmatrix}_0^R = \begin{Bmatrix} 1.0 \\ -1 \times 10^6 \end{Bmatrix}$$

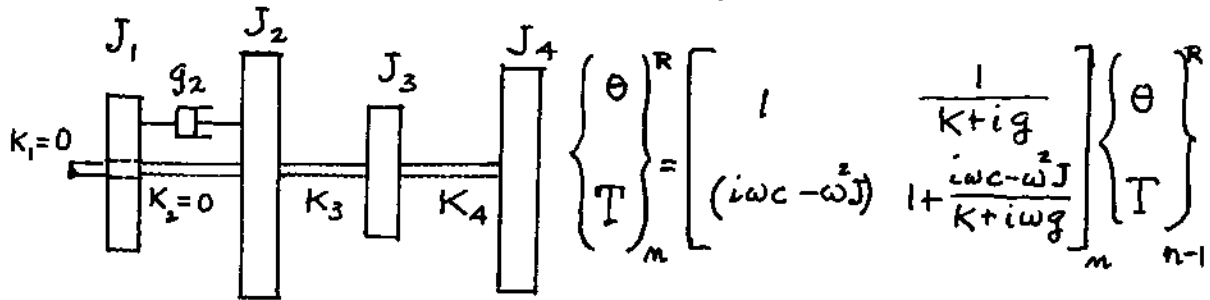
$$\begin{Bmatrix} \theta \\ T \end{Bmatrix}_1^R = \begin{bmatrix} 1 & -i(.446 \times 10^{-6}) \\ -5 \times 10^6 & (1 + i 2.23) \end{bmatrix} \begin{Bmatrix} 1.0 \\ -1 \times 10^6 \end{Bmatrix} = \begin{Bmatrix} 1 + .446 i \\ (-6 - i 2.23) \times 10^6 \end{Bmatrix}$$

$$\begin{Bmatrix} \theta \\ T \end{Bmatrix}_2^R = \begin{bmatrix} 1 & .2 \times 10^{-6} \\ -.5 \times 10^6 & .90 \end{bmatrix} \begin{Bmatrix} 1 + .446 i \\ (-6 - i 2.23) \times 10^6 \end{Bmatrix} = \begin{Bmatrix} -.20 \\ (-5.9 - i 2.23) \times 10^6 \end{Bmatrix}$$

$$\begin{Bmatrix} \theta \\ T \end{Bmatrix}_3^R = \begin{bmatrix} 1 & .10 \times 10^{-6} \\ -1.5 \times 10^6 & .85 \end{bmatrix} \begin{Bmatrix} -.20 \\ (-5.9 - i 2.23) \times 10^6 \end{Bmatrix} = \begin{Bmatrix} -.79 - i .223 \\ (-4.72 - i 1.9) \times 10^6 \end{Bmatrix}$$

Repeat for other frequencies - should be programmed for digital computer.





n	J	K	C	g
1	20.000	0.0000E 00	0.0000E 00	0.0000E 00
2	100.000	0.0000E 00	0.0000E 00	0.1000E 05
3	10.000	0.5000E 07	0.0000E 00	0.0000E 00
4	30.000	0.1000E 08	0.0000E 00	0.0000E 00

```

PROBLEM 10.21 THOMSON
DIMENSION RJ(4),RK(4),RC(4),RG(4),W(60)
COMPLEX DE(60,4),T(60,4),CRJ(4),CRK(4),CMPLX
M=4
L=60
READ,(RJ(J),J=1,M)
READ,(RK(J),J=1,M)
READ,(RC(J),J=1,M)
READ,(RG(J),J=1,M)
DO 20 I=1,L
  W(I)=I*10
  DO 5 J=1,4
    ZR=-W(I)**2*RJ(J)
    ZI=W(I)*RC(J)
    CRJ(J)=CMPLX(ZR,ZI)
    ZR=RK(J)
    ZI=W(I)*RG(J)
    CRK(J)=CMPLX(ZR,ZI)
5 CONTINUE
  DE(I,1)=1.
  T(I,1)=CRJ(1)*DE(I,1)
  DO 10 J=2,M
    DE(I,J)=DE(I,J-1)+T(I,J-1)/CRK(J)
    T(I,J)=CRJ(J)*DE(I,J-1)+(1.+CRJ(J)/CRK(J))*T(I,J-1)
10 CONTINUE
20 CONTINUE
  DO 25 J=1,M
    PRINT24,J,RJ(J),RK(J),RC(J),RG(J)
24 FORMAT(20X,I3,5X,F8.3,5X,E12.4,5X,E12.4,5X,E12.4)
25 CONTINUE
  DO 40 I=1,L
    PRINT30,W(I),DE(I,M),T(I,M)
30 FORMAT(10X,F8.2,10X,2E12.4,10X,2E12.4)
40 CONTINUE
STOP
END

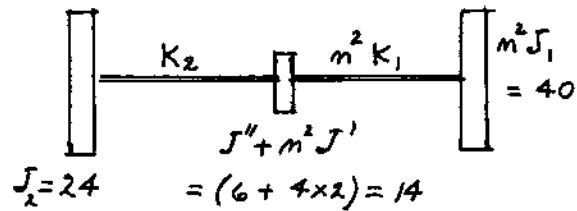
```

12-16 Cont.

ω	θ_{real}	θ_{imag}	T_{real}	T_{imag}
10.00	0.9963E 00	0.1994E-01	-0.1599E 05	-0.2798E 03
20.00	0.9852E 00	0.3950E-01	-0.6378E 05	-0.2233E 04
30.00	0.9667E 00	0.5833E-01	-0.1429E 06	-0.7505E 04
40.00	0.9409E 00	0.7604E-01	-0.2525E 06	-0.1769E 05
50.00	0.9077E 00	0.9226E-01	-0.3916E 06	-0.3429E 05
60.00	0.8671E 00	0.1066E 00	-0.5585E 06	-0.5873E 05
70.00	0.8193E 00	0.1188E 00	-0.7517E 06	-0.9225E 05
80.00	0.7642E 00	0.1284E 00	-0.9689E 06	-0.1360E 06
90.00	0.7019E 00	0.1350E 00	-0.1208E 07	-0.1908E 06
100.00	0.6324E 00	0.1384E 00	-0.1466E 07	-0.2575E 06
110.00	0.5558E 00	0.1381E 00	-0.1740E 07	-0.3365E 06
120.00	0.4722E 00	0.1339E 00	-0.2026E 07	-0.4280E 06
130.00	0.3816E 00	0.1253E 00	-0.2322E 07	-0.5320E 06
140.00	0.2840E 00	0.1120E 00	-0.2623E 07	-0.6480E 06
150.00	0.1797E 00	0.9379E-01	-0.2925E 07	-0.7754E 06
160.00	0.6853E-01	0.7024E-01	-0.3223E 07	-0.9131E 06
170.00	-0.4925E-01	0.4107E-01	-0.3514E 07	-0.1060E 07
180.00	-0.1736E 00	0.5975E-02	-0.3791E 07	-0.1213E 07
190.00	-0.3044E 00	-0.3535E-01	-0.4051E 07	-0.1372E 07
200.00	-0.4416E 00	-0.8320E-01	-0.4286E 07	-0.1532E 07
210.00	-0.5850E 00	-0.1378E 00	-0.4492E 07	-0.1692E 07
220.00	-0.7346E 00	-0.1996E 00	-0.4663E 07	-0.1847E 07
230.00	-0.8901E 00	-0.2686E 00	-0.4793E 07	-0.1993E 07
240.00	-0.1052E 01	-0.3452E 00	-0.4875E 07	-0.2126E 07
250.00	-0.1219E 01	-0.4297E 00	-0.4902E 07	-0.2241E 07
260.00	-0.1392E 01	-0.5222E 00	-0.4869E 07	-0.2332E 07
270.00	-0.1570E 01	-0.6229E 00	-0.4768E 07	-0.2394E 07
280.00	-0.1753E 01	-0.7322E 00	-0.4593E 07	-0.2419E 07
290.00	-0.1942E 01	-0.8501E 00	-0.4336E 07	-0.2400E 07
300.00	-0.2136E 01	-0.9768E 00	-0.3990E 07	-0.2331E 07
310.00	-0.2334E 01	-0.1113E 01	-0.3547E 07	-0.2201E 07
320.00	-0.2537E 01	-0.1257E 01	-0.3001E 07	-0.2004E 07
330.00	-0.2745E 01	-0.1412E 01	-0.2344E 07	-0.1729E 07
340.00	-0.2956E 01	-0.1575E 01	-0.1568E 07	-0.1367E 07
350.00	ω -0.3172E 01	-0.1748E 01	-0.6651E 06	-0.9071E 06
360.00	-0.3392E 01	-0.1931E 01	0.3715E 06	-0.3387E 06
370.00	-0.3616E 01	-0.2123E 01	0.1550E 07	0.3497E 06
380.00	-0.3842E 01	-0.2325E 01	0.2877E 07	0.1170E 07
390.00	-0.4072E 01	-0.2537E 01	0.4362E 07	0.2135E 07
400.00	-0.4306E 01	-0.2758E 01	0.6011E 07	0.3256E 07
410.00	-0.4542E 01	-0.2990E 01	0.7832E 07	0.4549E 07
420.00	-0.4780E 01	-0.3231E 01	0.9832E 07	0.6025E 07
430.00	-0.5021E 01	-0.3481E 01	0.1202E 08	0.7700E 07
440.00	-0.5264E 01	-0.3742E 01	0.1440E 08	0.9588E 07
450.00	-0.5508E 01	-0.4012E 01	0.1698E 08	0.1170E 08
460.00	-0.5755E 01	-0.4291E 01	0.1977E 08	0.1406E 08
470.00	-0.6002E 01	-0.4580E 01	0.2277E 08	0.1668E 08
480.00	-0.6251E 01	-0.4877E 01	0.2599E 08	0.1958E 08
490.00	-0.6500E 01	-0.5184E 01	0.2944E 08	0.2276E 08
500.00	-0.6750E 01	-0.5500E 01	0.3312E 08	0.2625E 08
510.00	-0.7000E 01	-0.5824E 01	0.3704E 08	0.3006E 08
520.00	-0.7250E 01	-0.6157E 01	0.4121E 08	0.3422E 08
530.00	-0.7500E 01	-0.6498E 01	0.4562E 08	0.3873E 08
540.00	-0.7748E 01	-0.6846E 01	0.5028E 08	0.4361E 08

12-18

After reduction to equivalent single shaft system, the freq. eq. of the degenerate 3-DOF system may be used.



$$\omega^4 - \left[\frac{K_1}{J_1} + \frac{K_2}{J_2} \left(1 + \frac{K_1}{K_2} + \frac{J_2}{J_3} \right) \right] \omega^2 + \frac{K_1}{J_1} \frac{K_2}{J_2} \left(\frac{J_1 + J_2 + J_3}{J_3} \right) = 0$$

$$\omega^4 - \left[\left(\frac{62.8}{24} \right) + \frac{59.7}{14} \left(1 + \frac{62.8}{59.7} + \frac{14}{40} \right) \right] 10^4 \omega^2 + \left[\frac{62.8}{24} \cdot \frac{59.7}{14} \left(\frac{24 + 14 + 40}{40} \right) \right] 10^8 = 0$$

$$\omega^4 - 12.82 \times 10^4 \omega^2 + 21.8 \times 10^6 = 0$$

$$\omega^2 = \begin{cases} 10.86 \times 10^4 \\ 2.01 \times 10^4 \end{cases} \quad \omega = \begin{cases} 329.2 \text{ r/s} \\ 141.8 \text{ "} \end{cases} = \begin{cases} 52.30 \text{ Hz} \\ 22.55 \text{ "} \end{cases}$$

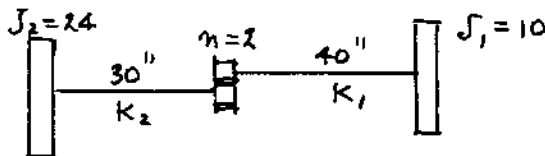
12-17

$$\omega^2 = \frac{(J_2 + m^2 J_1) K_2 (m^2 K_1)}{(m^2 J_1) J_2 (K_2 + m^2 K_1)}$$

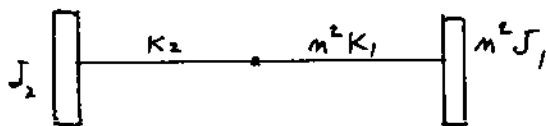
$$K_1 = G \frac{\pi D^4}{32 l} = \frac{(12 \times 10^6) \pi (1.5)^4}{32 \times 40} = 0.1492 \times 10^6$$

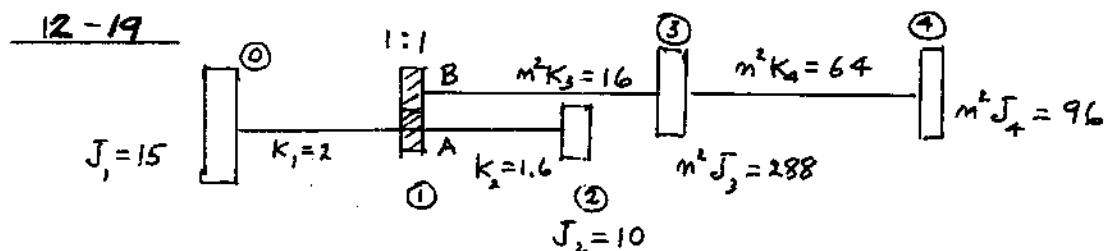
$$K_2 = \frac{(12 \times 10^6) \pi (2)^4}{32 \times 30} = 0.628 \times 10^6$$

$$\omega^2 = \frac{(24 + 40)(.628)(.1492) \times 10^{12}}{(240)(.628 + .5975) \times 10^6} = 2.04 \times 10^4$$



$$\omega = 22.7 \text{ Hz}$$





J & K along shaft B are multiplied by $m^2 = 16$ so that the gear ratio is reduced to 1:1

Then $\theta_{A1}^R = -\theta_{B1}^R = \theta_1^L$ and $T_{A1}^R + T_{B1}^R = T_1^L$

$$\begin{Bmatrix} \theta \\ T \end{Bmatrix}_0^R = \begin{Bmatrix} 1 \\ -15\omega^2 \end{Bmatrix} = \text{starting eq.}$$

$$\begin{Bmatrix} \theta \\ T \end{Bmatrix}_1^L = \begin{bmatrix} 1 & .5 \times 10^{-6} \\ 0 & 1 \end{bmatrix} \begin{Bmatrix} 1 \\ -15\omega^2 \end{Bmatrix} = \begin{Bmatrix} 1 - 7.5 \times 10^{-6} \omega^2 \\ -15\omega^2 \end{Bmatrix}$$

$$\begin{Bmatrix} \theta \\ T \end{Bmatrix}_3^R = \begin{bmatrix} 1 & .0625 \times 10^{-6} \\ -288\omega^2 & (1 - 18 \times 10^{-6} \omega^2) \end{bmatrix} \begin{Bmatrix} \theta \\ T \end{Bmatrix}_{B1}^R$$

$$\begin{Bmatrix} \theta \\ T \end{Bmatrix}_4^R = \begin{bmatrix} 1 & .01562 \times 10^{-6} \\ -96\omega^2 & (1 - 1.5 \times 10^{-6} \omega^2) \end{bmatrix} \begin{Bmatrix} \theta \\ T \end{Bmatrix}_3^R$$

$$\begin{Bmatrix} \theta \\ T \end{Bmatrix}_2^R = \begin{bmatrix} 1 & .625 \times 10^{-6} \\ -10\omega^2 & (1 - 6.25 \times 10^{-6} \omega^2) \end{bmatrix} \begin{Bmatrix} \theta \\ T \end{Bmatrix}_{A1}^R$$

Solution

$$\omega_1 = 377.2 \quad \begin{Bmatrix} \theta_1 \\ \theta_2 \end{Bmatrix}^{(1)} = \begin{Bmatrix} 1.0 \\ -4.35 \end{Bmatrix}$$

$$\omega_2 = 427.0 \quad \begin{Bmatrix} \theta_1 \\ \theta_2 \end{Bmatrix}^{(2)} = \begin{Bmatrix} 1.0 \\ 2.605 \end{Bmatrix}$$

$$\omega_3 = 940.0 \quad \begin{Bmatrix} \theta_1 \\ \theta_2 \end{Bmatrix}^{(3)} = \begin{Bmatrix} 1.0 \\ 1.725 \end{Bmatrix}$$

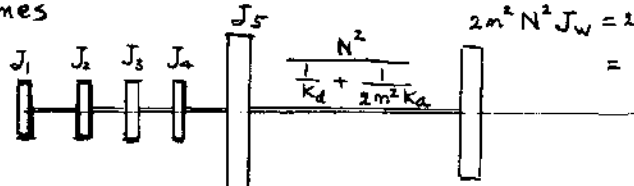
12-20

With $n = \frac{1}{3.5}$ the rear wheels and axle

are replaced by $2n^2 J_w$ and $2n^2 K_a$. In series with the drive shaft the stiffness is

$$\frac{1}{\frac{1}{K_d} + \frac{1}{2n^2 K_a}} \quad \text{Let } N = \frac{1}{3} = \text{transmission gear ratio,}$$

and refer all stiffnesses and inertias to engine speed by multiplying by N^2 . System then becomes



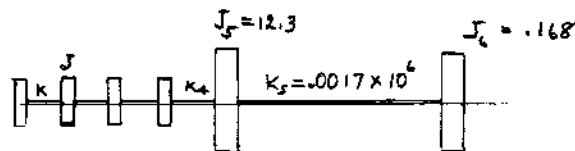
$$2n^2 N^2 J_w = 2\left(\frac{1}{3.5}\right)^2 \left(\frac{1}{3}\right)^2 9.2 = .167$$

$$K_a = \frac{(12 \times 10^6) \pi (1.25)^4}{32 \times 25} = .115 \times 10^6$$

$$K_d = \frac{(12 \times 10^6) \pi (1.5)^4}{32 \times 74} = .0806 \times 10^6$$

$$K_s = \frac{\left(\frac{1}{3}\right)^4 \times 10^6}{\frac{1}{.0806} + \frac{3.5^2}{.115}} = .00170 \times 10^6$$

12-21



$$J = J_1 = J_2 = J_3 = J_4 = 0.20 \text{ lb in sec}^2$$

$$K = 6.1 \times 10^6 \text{ lb in/rad}$$

$$K_s = 4.5 \times 10^6 \text{ lb in/rad}$$

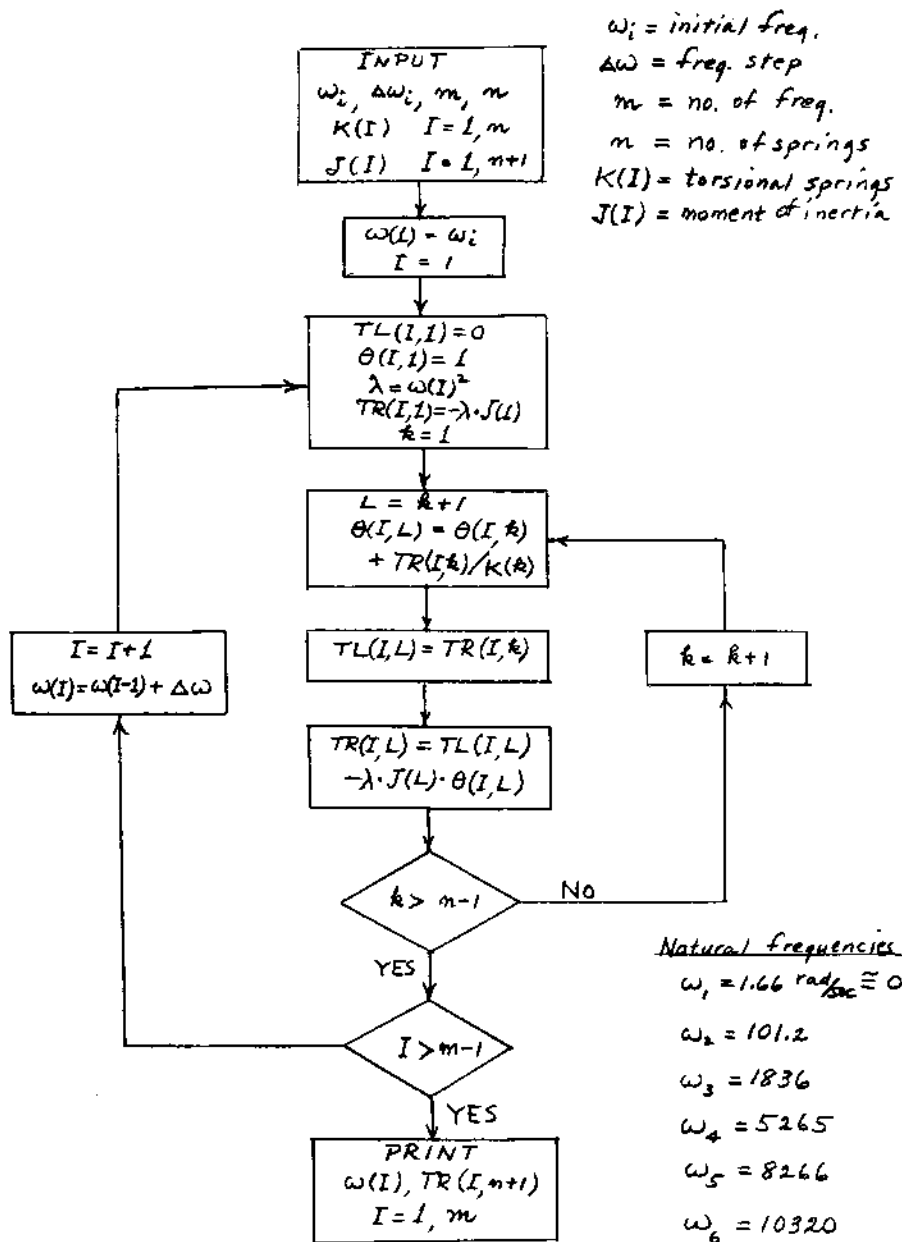
Approximation to first nat. freq. as two mass system

$$\omega = \sqrt{\frac{K(J_5 + J_6)}{J_1 J_2}} = 10^3 \sqrt{\frac{.0017(13.1 + .168)}{13.1 \times .168}}$$

$$= 10^3 \sqrt{.0102} \approx 100 \text{ rad/sec.}$$

computer solution follows

12-21 Cont.



Flow diagram for digital computation of
Prob 12-21

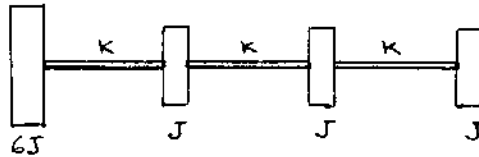
FORTRAN IV G LEVEL 21

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    DIMENSION SJ(6),SK(5),THETA(2000,6),TR(2000,6),TL(2000,6),
    S OMEG(2000)
    M=2000
    CM=50.
    STEP=20.
    CC 10 I=1,3
    SJ(I)=.2
10  SK(I)=0.61E 07
    SJ(4)=.2
    SJ(5)=12.3
    SJ(6)=.168
    SK(4)=0.45E 07
    SK(5)=0.17E 04
    CC 40 I=1,M
    TL(I,1)=0.
    THETA(I,1)=1.
    CMSQ=CM**2
    TR(I,1)=-CMSQ*SJ(1)
    CC 30 K=1,5
    L=K+1
    THETA(I,L)=THETA(I,K)+TR(I,K)/SK(K)
    TL(I,L)=TR(I,K)
30  TR(I,L)=TL(I,L)-CMSQ*SJ(L)*THETA(I,L)
    OMEG(I)=OM
40  CM=CM+STEP
    N=M/2
    PRINT 50
    PRINT 60,((OMEG(I),TR(I,6),OMEG(I+N),TR(I+N,6)),I=1,N)
50  FORMAT('1',' OMEGA',10X,'TR(CM,6)',33X,'OMEGA',10X,'TR(OM,6)')
60  FORMAT('1'.F7.1,5X,E15.4,30X,F7./, 5X,E15.4)
    STOP
    END

```

12-22



The system is first reduced to the above model

The equations in matrix form are

$$K \begin{bmatrix} -1 & 1 & 0 & 0 \\ 1 & -2 & 1 & 0 \\ 0 & 1 & -2 & 1 \\ 0 & 0 & 1 & -1 \end{bmatrix} \begin{Bmatrix} \theta_1 \\ \theta_2 \\ \theta_3 \\ \theta_4 \end{Bmatrix} = -\omega^2 J \begin{bmatrix} 1 & 0 & 0 & 0 \\ 0 & 1 & 0 & 0 \\ 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 6 \end{bmatrix} \begin{Bmatrix} \theta_1 \\ \theta_2 \\ \theta_3 \\ \theta_4 \end{Bmatrix}$$

Rearrange to

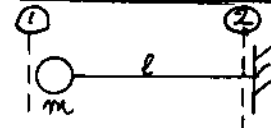
$$\frac{K}{\omega^2 J} \begin{bmatrix} 1 & -1 & 0 & 0 \\ -1 & 2 & -1 & 0 \\ 0 & -1 & 2 & -1 \\ 0 & 0 & -1.667 & 1.667 \end{bmatrix} \begin{Bmatrix} \theta_1 \\ \theta_2 \\ \theta_3 \\ \theta_4 \end{Bmatrix} = \begin{Bmatrix} \theta_1 \\ \theta_2 \\ \theta_3 \\ \theta_4 \end{Bmatrix}$$

Results of iteration with sweeping matrix

$\omega_0 = 0, \begin{Bmatrix} 1.0 \\ 1.0 \\ 1.0 \\ 1.0 \end{Bmatrix}$	$\omega_2 = 1.272 \sqrt{\frac{K}{J}}, \begin{Bmatrix} 1.00 \\ -1.614 \\ -1.237 \\ 1.416 \end{Bmatrix}$
$\omega_1 = .5375 \sqrt{\frac{K}{J}}, \begin{Bmatrix} 1.00 \\ .714 \\ .299 \\ -1.326 \end{Bmatrix}$	$\omega_3 = 1.825 \sqrt{\frac{K}{J}}, \begin{Bmatrix} 1.00 \\ -2.27 \\ 1.87 \\ -1.1005 \end{Bmatrix}$

12-23

From Eq. 12.9-5



$$\begin{Bmatrix} -V \\ M \\ 0 \\ 0 \end{Bmatrix}_2 = \begin{bmatrix} \vdots & \vdots & \vdots & \vdots \\ \vdots & \vdots & \vdots & \vdots \\ \vdots & \vdots & \vdots & \vdots \\ l & (1 + \frac{m\omega^2 l^3}{6EI}) \end{bmatrix} \begin{Bmatrix} 0 \\ 0 \\ \theta \\ 1 \end{Bmatrix}_1$$

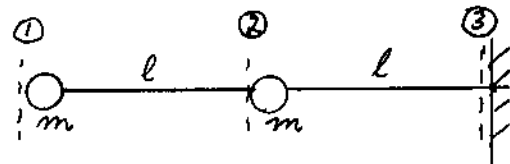
$$\therefore \begin{vmatrix} 1 & \frac{m\omega^2 l^2}{2EI} \\ l & (1 + \frac{m\omega^2 l^3}{6EI}) \end{vmatrix} = 0$$

$$\omega = \sqrt{\frac{3EI}{ml^3}}$$

12-24

From Eq. 12.9-5

Let $\alpha = l/6EI$



$$\begin{Bmatrix} -V \\ M \\ 0 \\ 0 \end{Bmatrix}_3 = \begin{bmatrix} 1 & 0 & 0 & m\omega^2 \\ l & 1 & 0 & m\omega^2 l \\ 3\alpha l & 6\alpha & 1 & 3m\omega^2 \alpha l \\ \alpha l^2 & 3\alpha l & l & (1+m\omega^2 \alpha l^2) \end{bmatrix}^2 \begin{Bmatrix} 0 \\ 0 \\ \theta \\ 1 \end{Bmatrix}_1$$

We need only to calculate the last two columns of the last two rows, which are:

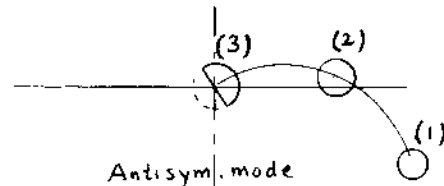
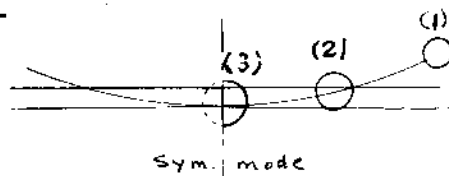
$$\begin{bmatrix} (1 + 3m\omega^2 \alpha l^2) & [15m\omega^2 \alpha l + 3(m\omega^2 \alpha)^2 l^3] \\ (2l + m\omega^2 \alpha l^3) & [1 + 9m\omega^2 \alpha l^2 + (m\omega^2 \alpha l^2)^2] \end{bmatrix} \begin{Bmatrix} \theta \\ 1 \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \end{Bmatrix}$$

Det. set to zero gives

$$(m\omega^2 \alpha l^2)^2 - 3(m\omega^2 \alpha l^2) + \frac{1}{6} = 0$$

Solving $\omega = \begin{cases} 0.583 \sqrt{\frac{EI}{m l^3}} \\ 4.20 \sqrt{\frac{EI}{m l^3}} \end{cases}$

12-25



$$\begin{Bmatrix} 0 \\ M \\ 0 \\ y \end{Bmatrix}_3 = \begin{bmatrix} u_{ij} \end{bmatrix} \begin{Bmatrix} 0 \\ 0 \\ \theta \\ y \end{Bmatrix}_1$$

Freq. eq.

$$\begin{vmatrix} u_{13} & u_{14} \\ u_{33} & u_{34} \end{vmatrix} = 0$$

$$\begin{Bmatrix} -V \\ 0 \\ \theta \\ 0 \end{Bmatrix}_3 = \begin{bmatrix} u_{ij} \end{bmatrix} \begin{Bmatrix} 0 \\ 0 \\ \theta \\ y \end{Bmatrix}_1$$

Freq. eq.

$$\begin{vmatrix} u_{23} & u_{24} \\ u_{43} & u_{44} \end{vmatrix} = 0$$

12-26



$$-J\omega^2 \theta_n = K(\theta_{n-1} - \theta_n) - K(\theta_n - \theta_{n+1})$$

$$\theta_{n-1} - 2\left(1 - \frac{\omega^2 J}{2K}\right)\theta_n + \theta_{n+1} = 0 \quad \text{let } \theta_n = e^{\lambda n}$$

$$e^{-\lambda} - 2\left(1 - \frac{\omega^2 J}{2K}\right) + e^{\lambda} = 0$$

$$\therefore \frac{\omega^2 J}{K} = 2(1 - \cosh \lambda) \quad \lambda = i\beta$$

$$= 2(1 - \cos \beta) \quad \theta_n = e^{i\beta n} = \cos \beta n + i \sin \beta n$$

Assume sol. $\theta_n = A \sin \beta n + B \cos \beta n$

Boundary $\theta_0 = 0 \quad \therefore B = 0$ and $\theta_n = A \sin \beta n$

Boundary N $-\omega^2 J \theta_N = -K(\theta_N - \theta_{N-1})$

$$\left(1 - \frac{J\omega^2}{K}\right)\theta_N = \theta_{N-1}$$

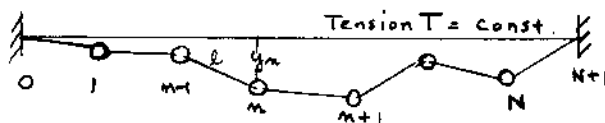
$$[1 - 2(1 - \cos \beta)] \sin \beta N = \sin \beta(N-1)$$

Reduces to $2 \sin \beta N \cos \beta = \sin \beta(N-1) + \sin \beta N$

$$2 \cos \beta(N + \frac{1}{2}) \sin \frac{\beta}{2} = 0 \quad \therefore \beta = 0, 2\pi, \dots$$

$$\omega_k = 2 \sqrt{\frac{K}{J}} \sin \frac{(2k-1)\pi}{2(2N+1)} \quad k = 1, 2, 3, \dots, N$$

12-27



$$m \ddot{y}_n = -\frac{T}{l}(y_n - y_{n-1}) + \frac{T}{l}(y_{n+1} - y_n) \quad \text{rearrange to}$$

$$y_{n+1} - 2\left(1 - \frac{\omega^2 m l}{2T}\right)y_n + y_{n-1} = 0$$

let $y_n = e^{i\beta n}$ & subst. into above eq.

$$e^{i\beta(n+1)} - 2\left(1 - \frac{\omega^2 m l}{2T}\right)e^{i\beta n} + e^{i\beta(n-1)} = 0$$

12-27 Cont:

$$1 - \frac{\omega^2 m l}{2T} = \frac{e^{i\beta} + e^{-i\beta}}{2} = \cos \beta$$

$$\frac{\omega^2 m l}{2T} = 1 - \cos \beta = 2 \sin^2 \frac{\beta}{2}$$

freq. eq. $\omega = 2 \sqrt{\frac{T}{ml}} \sin \frac{\beta}{2}$ Evaluate β at boundaries starting with gen. sol.

$$y_m = A \cos \beta m + B \sin \beta m$$

$$y_0 = 0 \quad \therefore A = 0, \quad y_{N+1} = 0 \quad \therefore \sin \beta(N+1) = 0$$

$$\beta(N+1) = 0, \pi, 2\pi, 3\pi, \dots = k\pi$$

$$\therefore \omega_k = 2 \sqrt{\frac{T}{ml}} \sin \frac{k\pi}{2(N+1)} \quad k = 1, 2, 3, \dots$$

For $N=2$

$$\omega_1 = 2 \sqrt{\frac{T}{ml}} \sin \frac{\pi}{6} = \sqrt{\frac{T}{ml}}$$

$$\omega_2 = 2 \sqrt{\frac{T}{ml}} \sin \frac{2\pi}{6} = \sqrt{\frac{3T}{ml}}$$

12-28

$$m\ddot{x}_m = k(x_{m+1} - x_m) - k_m(x_m - x_{m-1})$$

$$x_{m+1} - 2\left(1 - \frac{\omega^2 m}{k}\right)x_m + x_{m-1} = 0$$

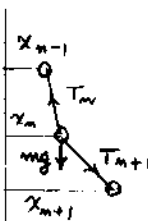
This problem is identical to Prob. 12-27 with changes in boundary conditions $\therefore \omega_k = 2 \sqrt{\frac{k}{m}} \sin \frac{k\pi}{2(N+1)}$ $k=1, 2, \dots$

12-29

$$m\ddot{x}_m = \frac{T_{m+1}}{l}(x_{m+1} - x_m) - \frac{T_m}{l}(x_m - x_{m-1})$$

$$T_m \equiv T'_{m+1} + mg$$

$$x_{m+1} - 2\left(1 - \frac{\omega^2 m l}{T_m}\right)x_m + x_{m-1} = \frac{mg l}{T_m}(x_{m+1} - x_m)$$



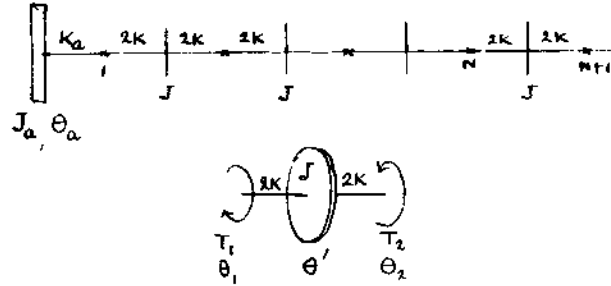
T_m changes i.e. $T_N = mg$, $T_{N-1} = 2mg$ etc.

\therefore Usual diff eq with const. coef. cannot be used.

Must solve as a system of N eqs.

$$-\omega^2 m x_1 = -\frac{Nmg}{l} x_1 + \frac{(N-1)mg}{l} (x_2 - x_1)$$

$$-\omega^2 m x_N = -\frac{mg}{l} (x_N - x_{N-1})$$



$$\begin{aligned}
 T_1 &= 2K(\theta' - \theta_1) & \rightarrow \begin{Bmatrix} T_1 \\ \theta_1 \end{Bmatrix} &= \begin{bmatrix} 1 & 0 \\ -\frac{1}{2K} & 1 \end{bmatrix} \begin{Bmatrix} T_1 \\ \theta' \end{Bmatrix} \\
 T_2 - T_1 &= -\omega^2 J \theta' & \rightarrow \begin{Bmatrix} T_1 \\ \theta' \end{Bmatrix} &= \begin{bmatrix} 1 & \omega^2 J \\ 0 & 1 \end{bmatrix} \begin{Bmatrix} T_2 \\ \theta' \end{Bmatrix} \\
 T_2 &= 2K(\theta_2 - \theta') & \rightarrow \begin{Bmatrix} T_2 \\ \theta' \end{Bmatrix} &= \begin{bmatrix} 1 & 0 \\ -\frac{1}{2K} & 1 \end{bmatrix} \begin{Bmatrix} T_2 \\ \theta_2 \end{Bmatrix}
 \end{aligned}$$

$$\begin{aligned}
 \therefore \begin{Bmatrix} T_1 \\ \theta_1 \end{Bmatrix} &= \begin{bmatrix} 1 & 0 \\ -\frac{1}{2K} & 1 \end{bmatrix} \begin{bmatrix} 1 & \omega^2 J \\ 0 & 1 \end{bmatrix} \begin{bmatrix} 1 & 0 \\ -\frac{1}{2K} & 1 \end{bmatrix} \begin{Bmatrix} T_1 \\ \theta_2 \end{Bmatrix} \\
 &= \begin{bmatrix} (1 - \frac{\omega^2 J}{2K}) & \omega^2 J \\ -(\frac{1}{K} - \frac{\omega^2 J}{4K}) & (1 - \frac{\omega^2 J}{2K}) \end{bmatrix} \begin{Bmatrix} T_2 \\ \theta_2 \end{Bmatrix} = \begin{bmatrix} A & B \\ C & A \end{bmatrix} \begin{Bmatrix} T_2 \\ \theta_2 \end{Bmatrix}
 \end{aligned}$$

Note that $A^2 - BC = 1$ $B = \frac{A^2 - 1}{C}$

Let $A = (1 - \frac{\omega^2 J}{2K}) = \cosh \lambda$

$C = -(\frac{1}{K} - \frac{\omega^2 J}{4K}) = \frac{1}{2} \sinh \lambda$

$B = \omega^2 J = \frac{\cosh^2 \lambda - 1}{\frac{1}{2} \sinh \lambda} = 2 \sinh \lambda$

$$\begin{Bmatrix} T_1 \\ \theta_1 \end{Bmatrix} = \begin{bmatrix} \cosh \lambda & 2 \sinh \lambda \\ \frac{1}{2} \sinh \lambda & \cosh \lambda \end{bmatrix} \begin{Bmatrix} T_2 \\ \theta_2 \end{Bmatrix} \quad \text{If replaced by exponentials it can be proved that}$$

$$\begin{bmatrix} \cosh \lambda & 2 \sinh \lambda \\ \frac{1}{2} \sinh \lambda & \cosh \lambda \end{bmatrix}^n = \begin{bmatrix} \cosh n\lambda & 2 \sinh n\lambda \\ \frac{1}{2} \sinh n\lambda & \cosh n\lambda \end{bmatrix}$$

$$\therefore \begin{Bmatrix} T_1 \\ \theta_1 \end{Bmatrix} = \begin{bmatrix} \cosh n\lambda & 2 \sinh n\lambda \\ \frac{1}{2} \sinh n\lambda & \cosh n\lambda \end{bmatrix} \begin{Bmatrix} T_{n+1} \\ \theta_{n+1} \end{Bmatrix}$$

From $A = (1 - \frac{\omega^2 J}{2K}) = \cosh \lambda$

$$\left. \begin{aligned} \omega^2 J = B = 2K(1 - \cosh \lambda) = -4K \sinh^2 \frac{\lambda}{2} \end{aligned} \right\} \therefore \text{freq. eq. } \omega = 2\sqrt{\frac{K}{J}} \sinh \frac{\lambda}{2}$$

where λ must be determined from boundary

12-30 Cont.

Boundary conditions

$$\theta_1 - \theta_a = \frac{T_1}{K_a} = -\frac{\omega^2 J_a \theta_a}{K_a}, \quad T_{N+1} = 0$$

$$\therefore T_1 = 2 \sinh N\lambda \cdot \theta_{N+1} = -\omega^2 J_a \theta_a$$

$$\theta_1 = \cosh N\lambda \cdot \theta_{N+1} = \left(1 - \frac{\omega^2 J_a}{K_a}\right) \theta_a$$

Divide $\frac{2 \sinh N\lambda}{\cosh N\lambda} = \frac{-\omega^2 J_a}{\left(1 - \frac{\omega^2 J_a}{K_a}\right)}$

$$\frac{2K(1 - \cosh \lambda)}{\sinh \lambda} \cdot \frac{\sinh N\lambda}{\cosh N\lambda} = \frac{-\omega^2 J_a}{\left(1 - \frac{\omega^2 J_a}{K_a}\right)}$$

$$-2K \left[\sinh N\lambda \cosh \lambda - \sinh N\lambda \right] = \frac{4K \frac{J_a}{J} \sinh^2 \frac{\lambda}{2}}{1 + \frac{4K \frac{J_a}{J} \sinh^2 \frac{\lambda}{2}}} \cdot \sinh \lambda \cosh N\lambda$$

$$\left(-2 \sinh N\lambda \cosh \lambda + 2 \sinh N\lambda \right) \left(1 - \frac{4K \frac{J_a}{J} \sinh^2 \frac{\lambda}{2}}{1 + \frac{4K \frac{J_a}{J} \sinh^2 \frac{\lambda}{2}} \right) = 2 \frac{J_a}{J} \sinh^2 \frac{\lambda}{2} \sinh \lambda \cosh N\lambda$$

$$\left(-\sin N\beta \cdot \cos \beta + \sin N\beta \right) \left(1 + \frac{4K \frac{J_a}{J} \sin^2 \frac{\beta}{2}}{1 + \frac{4K \frac{J_a}{J} \sin^2 \frac{\beta}{2}} \right) = -2 \frac{J_a}{J} \sin^2 \frac{\beta}{2} \sin \beta \cos N\beta$$

Solve for β & subst. into freq. eq.

$$\omega = 2 \sqrt{\frac{k}{J}} \sin \frac{\beta}{2}$$

This problem can also be solved by using stations at the disks instead of at points midway between disks. The equations

$$\theta_m = A \sin \beta m + B \cos \beta m, \quad m = 0, 1, 2, \dots, N$$

can be used provided $K_0 = \frac{1}{\frac{1}{K_a} + \frac{1}{2K}}$ is used.

12-31

With $m = 0$ fixed, solution is

$$X_m = B \sin \beta m$$

At top of the bldg, $m \ddot{x}_N = -k(x_N - x_{N-1}) - K_N x_N$

$$\text{or } X_{N-1} = \left(1 + \frac{K_N}{k} - \frac{m\omega^2}{k}\right) X_N$$

$$\therefore \sin \beta(N-1) = \left[1 + \frac{K_N}{k} - 2(1 - \cos \beta)\right] \sin \beta N$$

$$\sin \beta(N-1) - \sin \beta N + 2(1 - \cos \beta) \sin \beta N = \frac{K_N}{k} \sin \beta N$$

12-31 Cont.

$$\sin \beta N \cos \beta - \cos \beta N \sin \beta - \sin \beta N + 2 \sin \beta N - 2 \sin \beta N \cos \beta = \frac{K_N}{k} \sin \beta N$$

$$-\sin \beta N \cos \beta - \cos \beta N \sin \beta + \sin \beta N = \frac{K_N}{k} \sin \beta N$$

$$\therefore -\sin \beta(N+1) + \sin \beta N = \frac{K_N}{k} \sin \beta N$$

$$2 \cos \beta(N+\frac{1}{2}) \cdot \sin \frac{\beta}{2} = 2 [\cos \beta N \cos \frac{\beta}{2} \sin \frac{\beta}{2} - \sin \beta N \sin \frac{\beta}{2}] = [\cos \beta N \sin \beta - \sin \beta N (1 - \cos \beta)]$$

$$= \cos \beta N \sin \beta - \sin \beta N + \sin \beta N \cos \beta$$

$$\therefore -2 \cos \beta(N+\frac{1}{2}) \cdot \sin \frac{\beta}{2} = \frac{K_N}{k} \sin \beta N$$

12-32

System is same as those of Figs. P12-27 & P12-28

$$\therefore \omega_m = 2 \sqrt{\frac{k}{m}} \sin \frac{n\pi}{2(N+1)}$$

12-33

Let h = height between stories

$$m \ddot{y}_n = -k [(y_n - n h \theta) - (y_{n-1} - (n-1) h \theta)] + k [(y_{n+1} - (n+1) h \theta) - (y_n - n h \theta)]$$

$$= -k [y_n - y_{n-1} - h \theta] + k [y_{n+1} - y_n - h \theta]$$

$$= k [y_{n+1} - 2y_n + y_{n-1}]$$

\therefore for harmonic motion

$$Y_{n+1} - 2 \left(1 - \frac{m \omega^2}{2k}\right) Y_n + Y_{n-1} = 0$$

gen. sol. $Y_n = Y_0 \cos \beta n + B \sin \beta n$

boundary $Y_N = Y_0 \cos \beta N + B \sin \beta N$

$$B = \frac{Y_N - Y_0 \cos \beta N}{\sin \beta N}$$

Gen sol. becomes

$$Y_n = Y_0 \frac{\sin \beta(N-n)}{\sin \beta N} + Y_N \frac{\sin \beta n}{\sin \beta N}$$

boundary eq. for N^{th} mass

$$m \ddot{y}_N = -k [y_N - y_{N-1} + h \theta]$$

$$- \omega^2 m Y_N = -k [Y_N - Y_{N-1} + h \theta]$$

12-33 Cont:

$$\underline{\underline{(1 - \frac{\omega^2 m}{k}) Y_N = Y_{N-1} - l \theta}}$$

Torque Eq.

$$\sum_{n=1}^N m l (m \ddot{y}_n) - K_\theta \theta = (N+1) m \rho^2 \ddot{\theta}$$

$$-\omega^2 m l \sum_{n=1}^N n Y_n - K_\theta \theta = -(N+1) m \rho^2 \omega^2 \theta$$

or

$$\underline{\underline{\omega^2 m l \sum_{n=1}^N n Y_n - (K_\theta - (N+1) m \rho^2 \omega^2) \theta = 0}}$$

12-34

Flow diagram of FIG. 12.2-3 may be used
 From Prob. 12-3 range of ω may be from
 0 to 5000 ie $\omega_1 \approx 264$, $\omega_2 \approx 550$
 $\omega_3 < 5000$. We may choose $\Delta\omega = 20$ which
 will require 30 steps to get past ω_2 . For
 $\omega > 550$ it may be advisable to choose
 larger steps, ie $\Delta\omega = 100$ to locate ω_3
 after which a finer interval could be used.

Computer program may be written somewhat
 after that of Prob 12-3

12-35

Eq. 10-8 gives $\beta_k = \frac{(2k-1)\pi}{(2N+1)}$

Eq. 10-9 gives $\omega_k = 2\sqrt{\frac{k}{m}} \sin \frac{(2k-1)\pi}{2(2N+1)}$ $\left. \begin{array}{l} N = \text{DOF} \\ k = \text{mode number} \end{array} \right\}$

Then $X_m = B \sin \beta_k m$ where $m = \text{floor number}$

13-1

- (1) noise in radio reception - wide freq. range & probably stationary.
- (2) ground motion of earthquake - non stationary
- (3) ocean wave heights - non stationary and dependent on wind or sea state

13-2 see p

13-3 see p for definition = $\lim_{n \rightarrow \infty} \frac{1}{n} \sum_{i=1}^n x_i$

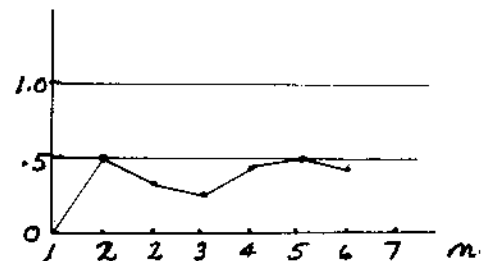
$E(h)$ for 800 coins thrown 100 times = 400

" " 8 " " " " = 4

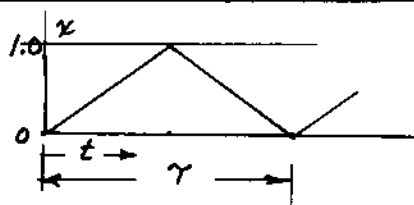
" " 8 " " 1000 " = 4

13-4 Make table as suggested here

Throw	H	T	sum	sum ÷ n
n = 1		0	0	0
2	1		1	1/2
3		0	1	1/3
4		0	1	1/4
5	1		2	2/5
6	1		3	3/6
7		0	3	3/7
etc				



13-5



$$x = \frac{2t}{\gamma} \quad t \leq \gamma/2$$

$$x = 2 - \frac{2t}{\gamma} \quad t \geq \gamma/2$$

Mean value = 0.50

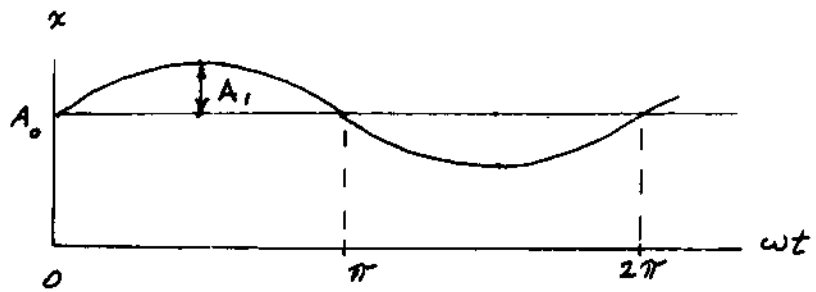
Mean square value

$$\bar{x}^2 = \frac{1}{\gamma} \left\{ \int_0^{\gamma/2} \left(\frac{2t}{\gamma} \right)^2 dt + \int_{\gamma/2}^{\gamma} \left[2 - \frac{2t}{\gamma} \right]^2 dt \right\}$$

$$= \frac{1}{\gamma} \left[\frac{4}{\gamma^2} \frac{t^3}{3} \right]_0^{\gamma/2} + \left(4t - \frac{8}{\gamma} \frac{t^2}{2} + \frac{4}{\gamma^2} \frac{t^3}{3} \right) \Big|_{\gamma/2}^{\gamma}$$

$$= \frac{1}{3}$$

13-6

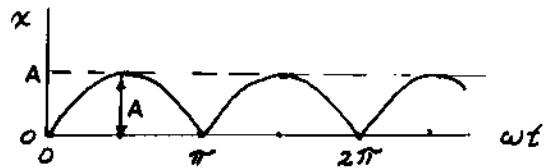


$x = A_0 + A_1 \sin wt$ - Mean val. = A_0 by inspection
Let $wt = 0$

$$\begin{aligned}\bar{x^2} &= \frac{1}{2\pi} \int_0^{2\pi} (A_0^2 + 2A_0A_1 \sin \theta + A_1^2 \sin^2 \theta) d\theta \\ &= \frac{1}{2\pi} \left\{ A_0^2 \theta - 2A_0A_1 \cos \theta + \frac{A_1^2}{2} \theta - \frac{A_1^2}{2} \frac{\sin 2\theta}{2} \right\}_0^{2\pi} \\ &= \frac{1}{2\pi} \left\{ A_0^2 2\pi - 2A_0A_1(1-1) + \frac{A_1^2}{2} 2\pi - 0 \right\} = A_0^2 + \frac{1}{2} A_1^2\end{aligned}$$

13-7

$$x = |A \sin wt|$$



$$\bar{x} = \frac{A}{\pi} \int_0^{\pi} \sin \theta d\theta = \frac{A}{\pi} (-\cos \theta) \Big|_0^{\pi} = \frac{2A}{\pi}$$

$$\bar{x^2} = \frac{A^2}{\pi} \int_0^{\pi} \sin^2 \theta d\theta = \frac{A^2}{\pi} \frac{\theta}{2} \Big|_0^{\pi} = \frac{A^2}{2}$$

13-8

Peak values are positive quantities \therefore cannot have a probability in the negative region. Also probability of zero or infinite peaks is zero.

13-9

Area under normal probability curve = 1.0

$$\int_{-\infty}^{\infty} \frac{e^{-\frac{x^2}{2\sigma^2}}}{\sigma \sqrt{2\pi}} dx = \frac{\sigma \sqrt{2\pi}}{\sigma \sqrt{2\pi}} = 1.0 \quad (a)$$

13-9 Cont.

For higher moments let $\alpha = \frac{1}{2\sigma^2}$ and

$$I = \int_{-\infty}^{\infty} e^{-\alpha x^2} dx \quad \dots \dots \dots (b)$$

From (a) $I = \sigma \sqrt{2\pi} = \sqrt{\pi} \alpha^{-\frac{1}{2}}$

Start with $I = \int_{-\infty}^{\infty} e^{-\alpha x^2} dx = \sqrt{\pi} \alpha^{-\frac{1}{2}} \quad (c)$

Differentiate w.r.t. α

$$\begin{aligned} \frac{\partial I}{\partial \alpha} &= - \int_{-\infty}^{\infty} x^2 e^{-\alpha x^2} dx = -\frac{1}{2} \sqrt{\pi} \alpha^{-\frac{3}{2}} \quad (d) \\ &= -\frac{1}{2} \sqrt{\pi} (2\sigma^2)^{\frac{3}{2}} = -\sqrt{2\pi} \sigma^3 \end{aligned}$$

$$\therefore \int_{-\infty}^{\infty} \frac{x^2 e^{-\alpha x^2}}{\sigma \sqrt{2\pi}} dx = \underline{\underline{\sigma^2}} = \underline{\underline{E(x^2)}}$$

Differentiate w.r.t. α again starting with (d)

$$\int_{-\infty}^{\infty} x^4 e^{-\alpha x^2} dx = -\frac{1}{2} \sqrt{\pi} \left\{ -\frac{3}{2} \alpha^{-\frac{5}{2}} \right\} = \frac{3}{4} \sqrt{\pi} (2\sigma^2)^{\frac{5}{2}}$$

$$\therefore \frac{1}{\sigma \sqrt{2\pi}} \int_{-\infty}^{\infty} x^4 e^{-\alpha x^2} dx = \underline{\underline{E(x^4) = 3\sigma^4}}$$

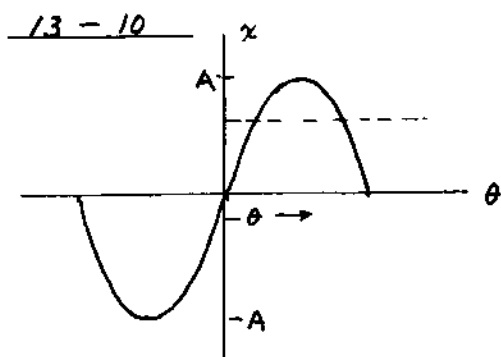
Repeat

$$\begin{aligned} \frac{\partial}{\partial \alpha} \left\{ \frac{3}{4} \sqrt{\pi} \alpha^{-\frac{5}{2}} \right\} &= -\frac{3}{4} \cdot \frac{5}{2} \sqrt{\pi} \alpha^{-\frac{7}{2}} = -\frac{3}{4} \cdot \frac{5}{2} \sqrt{\pi} (2\sigma^2)^{\frac{7}{2}} \\ &= -3 \cdot 5 \sqrt{2\pi} \sigma^7 \end{aligned}$$

\div by $\sigma \sqrt{2\pi}$ $\underline{\underline{E(x^6) = 3 \cdot 5 \sigma^6}}$

General Eq. is $\underline{\underline{E(x^n) = 1 \cdot 3 \cdot 5 \cdot \dots \cdot (n-1) \sigma^n}} \quad (\text{for } n \text{ even})$

by inspection $E(x^n) = 0$ for n odd.



$$x = A \sin \theta$$

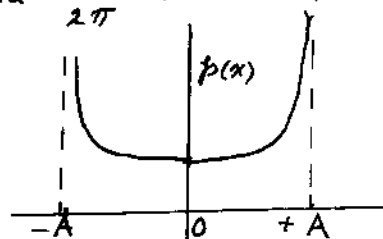
$$\text{When } x=0 \quad P(x) = \frac{1}{2}$$

i.e. half the time x is less than $x=0$.

As we increase x from $x=0$ we add $\frac{2\theta}{2\pi}$ to the probability

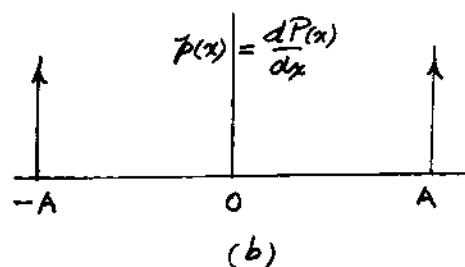
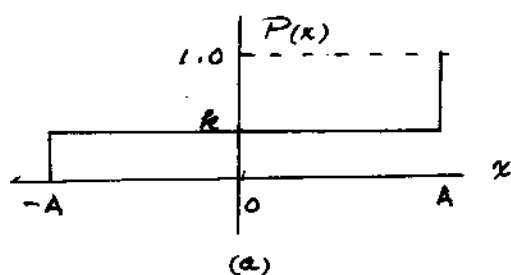
$$\theta = \sin^{-1} \frac{x}{A}$$

$$P(x) = \frac{1}{2} \pm \sin^{-1} \frac{x}{A}$$



$$p(x) = \frac{dP(x)}{dx} = \frac{1}{\pi} \frac{d}{dx} \left(\sin^{-1} \frac{x}{A} \right) = \frac{1}{\pi \sqrt{A^2 - x^2}}$$

13-11 Measure the total length of line at $-A$ and divide by total length at A and $-A$. Let this fraction be k ; then the cumulative prob. curve will appear as in (a). Density curve is shown in (b)



13-12 $x(t) = A \cos t$

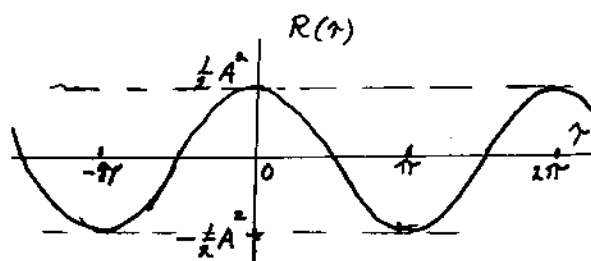
$$x(t+\gamma) = A \cos(t+\gamma) = A [\cos t \cos \gamma - \sin t \sin \gamma]$$

$$x(t)x(t+\gamma) = A^2 [\cos^2 t \cos \gamma - \cos t \sin t \sin \gamma]$$

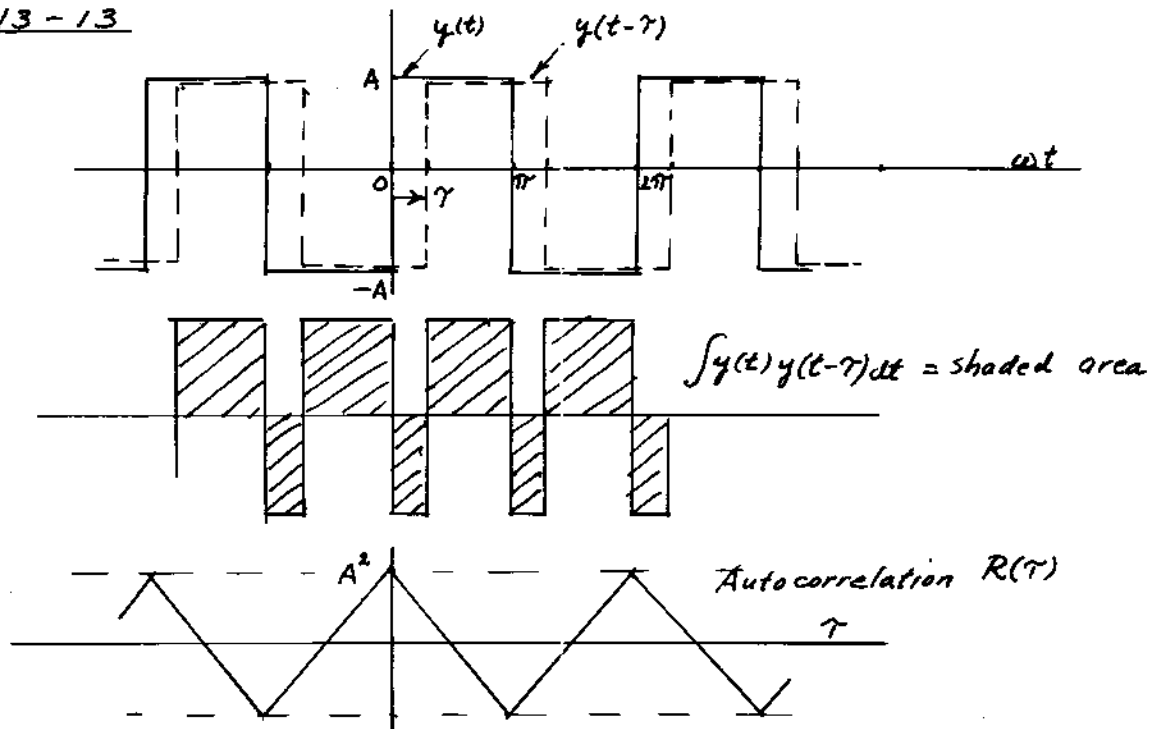
$$R(\gamma) = \lim_{T \rightarrow \infty} \frac{A^2}{T} \int_{-T/2}^{T/2} [\cos \gamma \cdot \frac{1}{2}(1 + \cos 2t) - \sin \gamma \cdot \sin t \cos t] dt$$

$$= \lim_{T \rightarrow \infty} \frac{A^2}{T} \left[\cos \gamma \cdot \left(\frac{T}{2} \right) - \sin \gamma \cdot (0) \right]$$

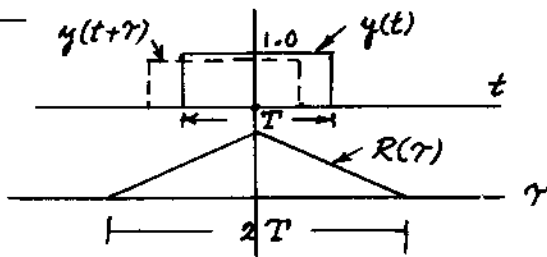
$$= \frac{A^2}{2} \cos \gamma$$



13-13

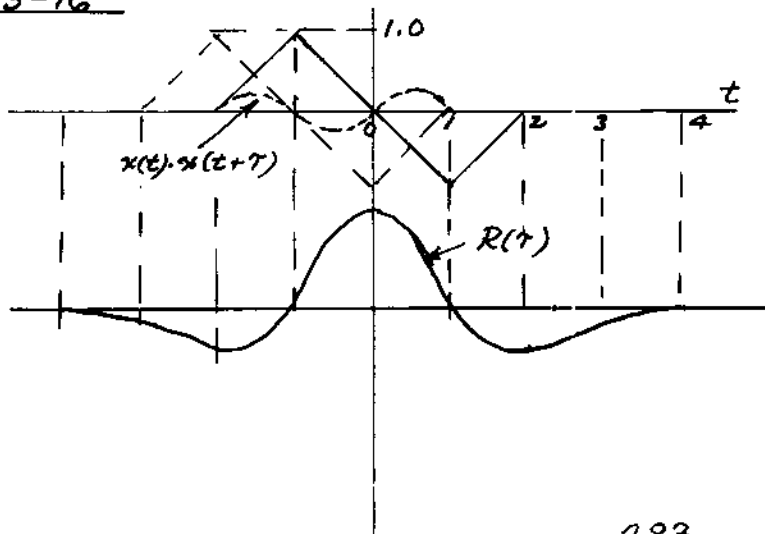


13-14



13-15 This problem is similar to Prob. 13-14. Integrate the area under $y(t) \cdot y(t+\tau)$ curve. Start curve with $R(\tau) = 5$ at $\tau = 0$ and linearly decrease to $R(\tau) = 1.0$ at $\tau = 1$, etc. Shift traced curve as suggested.

13-16



$$\underline{13-17} \quad \overline{x^2} = .20 \frac{g^2}{Hz} \times 500 Hz = 100 g^2$$

$$= 100 \times 9.81^2 = 9623.6$$

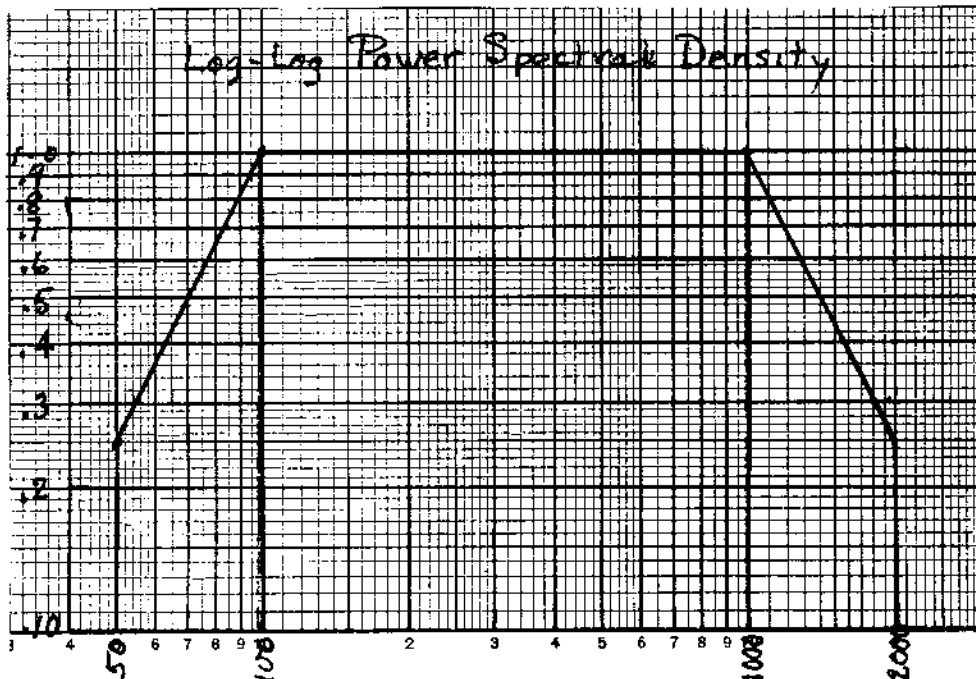
$$RMS = \bar{x} = \sqrt{9623.6} = 98.1 \text{ m/s}^2$$

13-18

$$Area = 1 \times 900 + 2 \times 1000 = 2900 g^2$$

$$RMS = 9.81 \sqrt{2900} = 528.3 \text{ m/s}^2$$

13-19

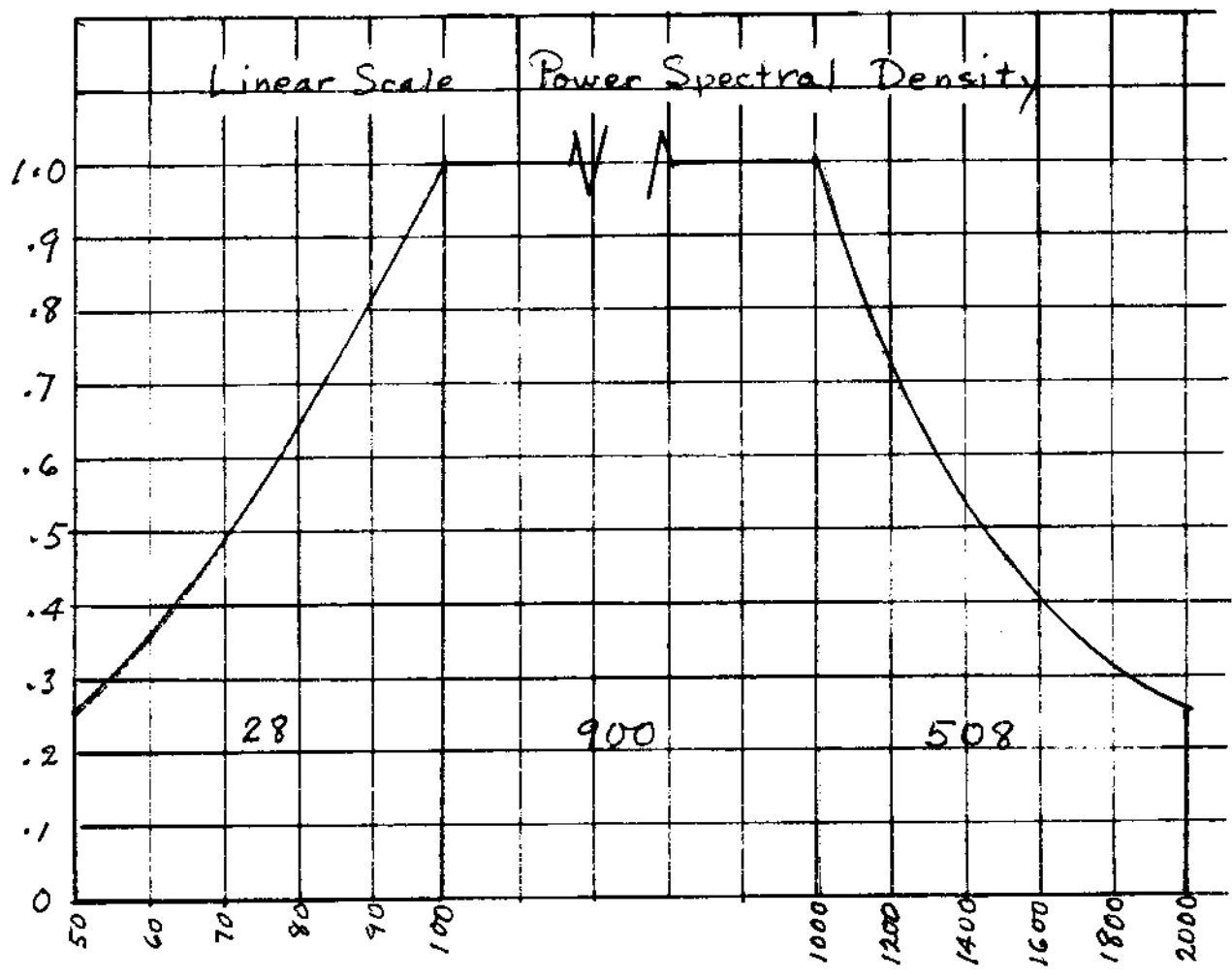


$$DB = 10 \log_{10} \frac{1.0}{.25} = 6.02 \text{ db/octave}$$

The log-log plot is replotted on linear scale

$$\overline{x^2} = \text{total area} = \underline{\underline{1436}} \quad RMS = 37.9 \text{ m/s}^2$$

13-19 Cont:



13-20 Procedure here is to determine the Fourier series and plot the quantity $\frac{1}{2} C_n C_n^*$ where $C_n = a_n - i b_n$. Note that $C_n C_n^* = a_n^2 + b_n^2$ i.e. for Fig. 13-20 (a), $x(t) = \frac{1}{2} - \frac{1}{\pi} [\sin \omega_1 t + \frac{1}{2} \sin 2\omega_1 t + \dots]$ (see prob. 1-12.) $\therefore a_n = 0, b_n = \frac{1}{n\pi}$

$$\sum \frac{1}{2} C_n C_n^* = \frac{1}{4} + \frac{1}{\pi^2} + \frac{1}{4\pi^2} + \frac{1}{9\pi^2} \dots$$



13-21

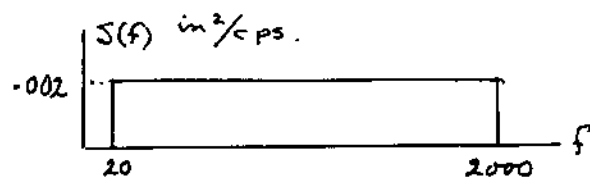
$$\overline{x^2} = .002 \times 1980 = 3.96 \text{ in}^2$$

$$\text{RMS} = \sqrt{\overline{x^2}} = 1.99 \text{ in}$$

$$\sigma^2 = \overline{x^2} - (\bar{x})^2$$

$$= 3.96 - (1.732)^2 = 0.9602$$

$$\sigma = 0.9799$$



$$\bar{x} = \sqrt{3.0} = 1.732''$$



13-22

C_n found by multiplying $f(t)$ by $e^{-in\omega_0 t}$ and integrating over one period

$$C_n = \frac{1}{T} \int_{-T/2}^{T/2} f(t) e^{-in\omega_0 t} dt$$

13-23

$$C_n = \frac{1}{2} (a_n - i b_n)$$

$$C_0 = \frac{1}{2} a_0$$

$$\text{Re}(C_n e^{in\omega_0 t}) = \frac{1}{2} (C_n e^{in\omega_0 t} + C_n^* e^{-in\omega_0 t})$$

$$\therefore f(t) = \frac{a_0}{2} + \frac{1}{2} \sum_{n=1}^{\infty} (C_n e^{in\omega_0 t} + C_n^* e^{-in\omega_0 t})$$

but $C_n^* = C_{-n} \therefore$

$$f(t) = \sum_{n=-\infty}^{\infty} \frac{1}{2} C_n e^{in\omega_0 t} = \sum_{n=-\infty}^{\infty} \rho_n e^{in\omega_0 t}$$

$$C_n = 2\rho_n \text{ (see Eq. 13.2-9)}$$

13-24

Same procedure as Prob 13-20

ie Fig P13-24 (See Prob 1-11)

$$x(t) = \frac{1}{2} + \frac{1}{\pi^2} \left(\cos \omega_1 t + \frac{1}{3^2} \cos 3\omega_1 t + \dots \right)$$

$$S(f) = \sum \frac{C_n C_n^*}{2} = \sum \bar{C}_n^2$$

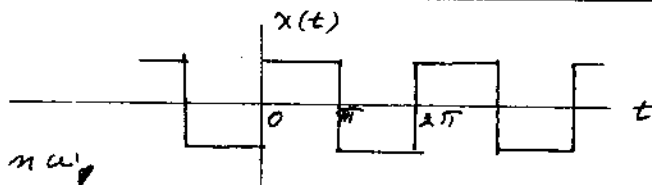
13-25

See Ch. 1 Sec. 1.2

13-26

$$x = \sum_{n=-\infty}^{\infty} C_n e^{i\omega_n t}$$

$$\omega_n = n\omega_1$$



$$C_n = \frac{1}{T} \int_{-T/2}^{T/2} x(t) e^{-i\omega_n t} dt = \frac{1}{2} (a_n - i b_n), \quad C_0 = \frac{A_0}{2}$$

Let $\omega_n t = n\omega_1 t = n\theta$ where $\begin{cases} \theta = \omega_1 t \\ d\theta = \omega_1 dt \end{cases}$

Limits $n\omega_1 \frac{T}{2} = n\theta$

$n \frac{2\pi}{T} \frac{T}{2} = n\pi = n\theta \quad \therefore \theta \text{ goes betw. } \pm \pi$

$$C_n = \frac{1}{T} \int_{-\pi}^{\pi} f(\theta) e^{-in\theta} \frac{d\theta}{\omega_1}$$

$$\omega_1 T = \frac{2\pi}{T}, \quad T = 2\pi$$

$$= \frac{1}{2\pi} \int_{-\pi}^{\pi} f(\theta) e^{-in\theta} d\theta = \frac{2A}{i n \pi} \quad \text{for } n \text{ odd}$$

$$= 0 \quad \text{for } n \text{ even}$$

$$\therefore x(t) = \frac{2A}{\pi} \sum_{n=-\infty}^{\infty} \frac{e^{in\omega_1 t}}{i n} = \frac{4A}{\pi} \sum_{n=0}^{\infty} \frac{1}{n} \left(\frac{e^{in\omega_1 t} - e^{-in\omega_1 t}}{2i} \right)$$

$$= \frac{4A}{\pi} \left[\sin \omega_1 t + \frac{1}{3} \sin 3\omega_1 t + \frac{1}{5} \sin 5\omega_1 t + \dots \right]$$

plot $\frac{1}{2}$ (square of ampl.)

$$\underline{13-27} \quad \left(\frac{x_k}{F}\right) = \frac{1}{\sqrt{(1-\eta^2)^2 + (25\eta)^2}} \quad \text{where } \eta = \frac{f}{f_m} = \frac{\omega}{\omega_m}$$

At $\eta = 1$ $\frac{x_k}{F} = \frac{1}{25}$ At half power pt. $\frac{x_k}{F} = \frac{1}{\sqrt{2}} \cdot \frac{1}{25}$

$$\therefore \frac{1}{2} \left(\frac{1}{25}\right)^2 = \frac{1}{(1-\eta^2)^2 + (25\eta)^2}$$

$$\eta^4 - 2\eta^2 + 1 + 45^2\eta^2 = 85^2$$

$$\eta^4 - 2(1-25^2)\eta^2 + (1-85^2) = 0$$

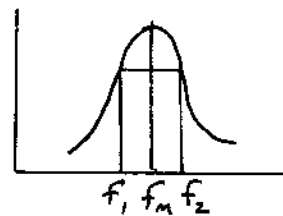
$$\therefore \eta^2 = (1-25^2) \pm 25\sqrt{1+5^2}$$

$$\therefore \eta \approx (1 \pm 25)^{1/2} = 1 \pm 5 + \text{negl. term}$$

$$\therefore f_1 = f_m(1-5) = f_m\left(1 - \frac{1}{2Q}\right)$$

$$f_2 = f_m(1+5) = f_m\left(1 + \frac{1}{2Q}\right)$$

$$\begin{cases} \text{for } \zeta \ll 1 \\ \eta^2 \approx 1 \pm 25 \\ \text{by neglecting } 5^2 \end{cases}$$



13-28

$$\int_0^{\infty} \frac{d\eta}{(1-\eta^2)^2 + (25\eta)^2}$$

where $\eta = \frac{f}{f_m}$

1st find poles from zeros of denominator

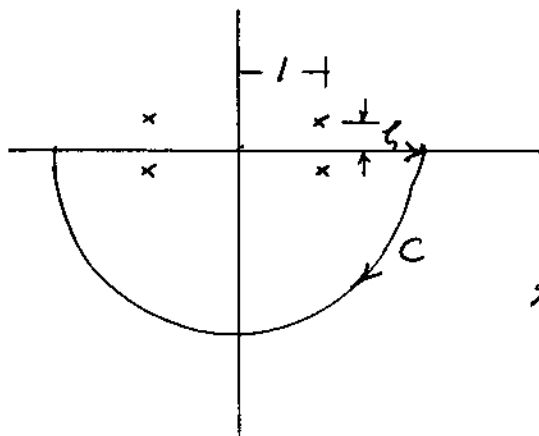
$$F(\eta) = \eta^4 - 2(1-25^2)\eta^2 + 1 = 0$$

$$\eta^2 = (1-25^2) \pm i 25\sqrt{1-5^2} \approx 1 \pm i 25$$

for $\zeta \ll 1$

See Theory of Residues p 135-137 Laplace Transformation
by W. T. Thomson - Prentice-Hall, Inc.

13-28 Cont:



Form contour with infinite circle. Poles are $\eta \approx 1 \pm i5$

Integrate around contour C

$$2 \int_0^{\infty} \dots \int_C = 2\pi i \sum \text{Residues within contour}$$

Residues at the two poles are

$$\frac{1}{F'(\eta)} = \frac{1}{2(1-\eta^2)(-2\eta) + 85^2 \eta} \quad \text{evaluated at } \eta_1 = 1-i5 = -(1+i5)$$

$$\approx i \frac{1}{45} \quad \text{and } \eta_1^2 = 1-i25$$

$$\eta_2^2 = 1+i25$$

Since $\int_C = 0$ we have

$$-2 \int_0^{\infty} = 2\pi i \left(\frac{i}{45} \right) = -\frac{\pi}{25}$$

$$\therefore \int_0^{\infty} \frac{d\eta}{(1-\eta^4)^2 + (25\eta)^2} = \frac{\pi}{45}$$

Can also be checked by numerical integration (must use very small $\Delta\eta$)

13-29 from definition

$$\bar{X}(s) = \frac{F(s)}{ms^2 + R(1+it)} = \bar{H}(s) \bar{F}(s)$$

13-30

$$H(\omega) = \frac{1}{k} \frac{1}{\sqrt{[1 - (\frac{\omega}{\omega_n})^2]^2 + [25 \frac{\omega}{\omega_n}]^2}}$$

Each component can be treated separately. For 1st component $F \cos(.5\omega_n t - \theta_1)$, the mean square response is

$$\frac{1}{[1 - (.5)^2]^2 + [.2 \times .5]^2} \times \frac{1}{2} \left(\frac{F}{k}\right)^2 = 1.746 \times \frac{1}{2} \left(\frac{F}{k}\right)^2$$

Similarly other components are

$$2^{\text{nd}} \text{ comp } \frac{1}{[.2]^2} \times \frac{1}{2} \left(\frac{F}{k}\right)^2 = 25 \times \frac{1}{2} \left(\frac{F}{k}\right)^2$$

$$3^{\text{rd}} \text{ comp } \frac{1}{[1 - 2^2]^2 + [.2 \times 2]^2} \times \frac{1}{2} \left(\frac{F}{k}\right)^2 = .109 \times \frac{1}{2} \left(\frac{F}{k}\right)^2$$

$$\therefore \bar{x}^2 = [1.746 + 25 + .109] \times \frac{1}{2} \left(\frac{F}{k}\right)^2 = 26.85 \times \frac{1}{2} \left(\frac{F}{k}\right)^2$$

$$= 13.43 \left(\frac{F}{k}\right)^2$$

13-31

From Sec 13.4 & Ex. 13.8-1, $\sigma = 26.6g$
 $2\sigma = 53.2g$

$$P[-2\sigma \leq x(t) \leq 2\sigma] = 95.4\%$$

$$\therefore P[x(t) \geq 2\sigma] = 100 - 95.4 = \underline{\underline{4.6\%}}$$

$$P[\bar{X} \geq 2\sigma] = \underline{\underline{13.5\%}}$$

13-32

$$m = 40 \text{ Kg.}$$

$$\omega_n = 2\pi \times 2.20 = 13.82$$

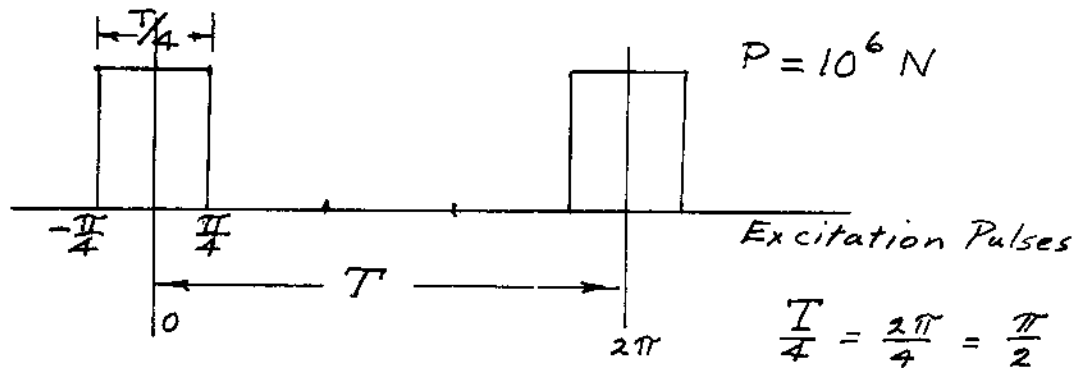
$$\text{Assume } \zeta = .15$$

$$k = m\omega_n^2 = 40 \times 13.82^2 = 7643.$$

$$f_n = 2.20 \text{ Hz}$$

$$\omega_1 = \frac{2\pi}{T}$$

$$X = \frac{F_0}{k} \frac{\sin(\omega t - \phi)}{\sqrt{[1 - (\frac{\omega}{\omega_n})^2]^2 + [2\zeta \frac{\omega}{\omega_n}]^2}}$$



F.S. of the excitation is a cosine series

$$a_n = \frac{2}{T} \int_{-\frac{T}{2}}^{\frac{T}{2}} x(t) \cos n\omega_1 t dt = \frac{1}{\pi} \int_{-\frac{\pi}{4}}^{\frac{\pi}{4}} P \cos n\theta \frac{d\theta}{\omega_1}$$

$$= \frac{P}{\pi\omega_1} \left. \frac{\sin n\theta}{n} \right|_{-\frac{\pi}{4}}^{\frac{\pi}{4}} = \left(\frac{P}{n\pi\omega_1} \right) 2 \sin \frac{n\pi}{4}$$

$$= \left(\frac{2P}{\pi\omega_1} \right) \left(\frac{\sin \frac{n\pi}{4}}{n} \right)$$

$$a_0 = \frac{2}{T} \int_{-\frac{T}{2}}^{\frac{T}{2}} P dt = \frac{P}{2}$$

n	$\frac{\sin \frac{n\pi}{4}}{n}$
1	$\sqrt{2}/2$
2	$1/2$
3	$\sqrt{2}/6$
4	0
5	$-\sqrt{2}/10$
6	$-1/6$

13-32 Cont:Excitation Spectrum $S_p(\omega)$

$$S_p(\omega) = \frac{a_0^2}{2} + \sum_{n=1}^{\infty} \frac{a_n^2}{2} = \frac{P^2}{8} + \sum_{n=1}^{\infty} \frac{1}{2} \left(\frac{2P}{\pi \omega_1} \right)^2 \frac{\sin^2 n \frac{\pi}{4}}{n^2}$$

$$= 10^{12} \left[\frac{1}{8} + \left(\frac{2}{\pi^2 \omega_1^2} \right) \left(\frac{1}{2} + \frac{1}{4} + \frac{1}{18} + 0 + \frac{1}{50} + \dots \right) \right]$$

Response Spectrum $\overline{y^2} = \int_0^{\infty} H H^* S_p df = \overline{x^2}$

$$= \sum H H^* S_p$$

$$H H^* = \frac{1}{k^2} \frac{1}{\left[1 - \left(\frac{\omega}{\omega_n} \right)^2 \right]^2 + \left[2.5 \frac{\omega}{\omega_n} \right]^2}$$

$$\omega = n \omega_1 = n \frac{2\pi}{T}$$

$$\omega_n = 2\pi f_n = 2\pi 2.2$$

$$\frac{\omega}{\omega_n} = n \left(\frac{1}{2.2 T} \right) \quad \text{Subst. into } \overline{x^2}$$

13-33 For base motion, the relative motion is given by Eq. 3.5-4, which can be written as

$$Z = \frac{\left(\frac{\omega}{\omega_n} \right)^2 Y}{\sqrt{\left[1 - \left(\frac{\omega}{\omega_n} \right)^2 \right]^2 + \left[2.5 \frac{\omega}{\omega_n} \right]^2}}$$

Thus with $S(f) = \text{Spectral density of excitation}$

$$\overline{Z^2} = \int_0^{\infty} S_Y(f_n) \frac{\left(\frac{f}{f_n} \right)^4 df}{\left[1 - \left(\frac{f}{f_n} \right)^2 \right]^2 + \left[2.5 \frac{f}{f_n} \right]^2} = S_Y(f_n) \cdot f_n \cdot \int_0^{\infty} \frac{\xi^4 d\xi}{\left[1 - \xi^2 \right]^2 + \left[2.5 \xi \right]^2}$$

\therefore must evaluate $\int_0^{\infty} \frac{\xi^4 d\xi}{\left[1 - \xi^2 \right]^2 + \left[2.5 \xi \right]^2}$

13-34

$$\ddot{x} = \frac{(k + i\omega c)}{k - m\omega^2 + i\omega c} \ddot{y} = \frac{1 + i(25 \frac{f}{f_m})}{1 - (\frac{f}{f_m})^2 + i(25 \frac{f}{f_m})} \ddot{y}$$

$$\begin{aligned} \overline{\ddot{x}^2} &= \int_0^\infty \overline{\ddot{y}^2} \frac{1 + i(25 \frac{f}{f_m})}{1 - (\frac{f}{f_m})^2 + i(25 \frac{f}{f_m})} \cdot \frac{1 - i(25 \frac{f}{f_m})}{1 - (\frac{f}{f_m})^2 - i(25 \frac{f}{f_m})} df \\ &= \int_0^\infty S_{\ddot{y}}(f) \cdot f_m \cdot \frac{1 + (25 \frac{f}{f_m})^2}{[1 - (\frac{f}{f_m})^2]^2 + [25 \frac{f}{f_m}]^2} \cdot d(\frac{f}{f_m}) \end{aligned}$$

13-35

$$\omega^2 = \frac{k}{m} = \frac{k}{60} = (2\pi \cdot 4)^2$$

$$k = 60 \times (8\pi)^2 = 37,899 \text{ N/m}$$

$$k^2 = 1436 \times 10^6 \text{ N}^2/\text{m}^2$$

$$H^2 = \frac{1}{1436 \times 10^6 \left\{ \left[1 - \left(\frac{\omega}{8\pi} \right)^2 \right]^2 + \left[2(0.05) \frac{\omega}{8\pi} \right]^2 \right\}}$$

$$\overline{y^2} = \int_0^\infty H^2 S(\omega) d\omega \approx \frac{S(\omega_m)}{k^2} f_m \frac{\pi}{45}$$

$$= \frac{100 \times 10^3}{1436 \times 10^6} \cdot \cancel{A} \cdot \frac{\pi}{\cancel{A} \times 0.05} = .00438$$

$$\sigma^2 = \overline{y^2} = .00438 \text{ m}^2, \quad \sigma = .0662 \text{ m}$$

$$y = .132 = 1.99 \sigma$$

$$P[y > 1.99\sigma] = \underline{\underline{4.6\%}}$$

13-36

$$m = 272 \text{ kg}$$

$$f_m = 26 \text{ Hz}$$

$$\omega_m = 2\pi f_m = 163.36$$

$$\zeta = 0.10$$

$$k = m\omega^2 = 272 \times (163.36)^2$$

$$= 7258 \times 10^3 \text{ N/m}$$

$$k^2 = 52.689 \times 10^{12}$$

$$\bar{x}^2 = \sigma^2 = \frac{S(f_m)}{k^2} \cdot f_m \cdot \frac{\pi}{4\zeta}$$

$$\sigma^2 = \frac{4 \times 10^6}{52.689 \times 10^{12}} \times 26 \times \frac{\pi}{4 \times 0.10} = 15.50 \times 10^{-6}$$

$$\sigma = .003937$$

$$0.012 \text{ m} = 3.05 \sigma$$

$$P[|x| > 3.05\sigma] = 0.3\%$$

13-37

$$S(f) = 5 \times 10^6 \text{ N}^2/\text{Hz}, \quad \zeta = 0.03$$

$$\omega_m = 30$$

$$f_m = \frac{\omega_m}{2\pi} = 4.775 \text{ Hz}$$

$$m = 1500 \text{ kg}, \quad k = m\omega^2 = 1500 \times 30^2 = 1.350 \times 10^6$$

$$k^2 = 1.823 \times 10^{12}$$

$$\sigma^2 = \bar{x}^2 = \frac{S(f_m)}{k^2} \cdot f_m \cdot \frac{\pi}{4\zeta} = \frac{5 \times 10^6}{1.823 \times 10^{12}} \times 4.775 \times \frac{\pi}{4 \times 0.03}$$

$$= 342.9 \times 10^{-6}$$

$$\sigma = 0.01852$$

$$.037 = 2\sigma$$

$$P[A > .037] = P[A > 2\sigma] = 13.5\%$$

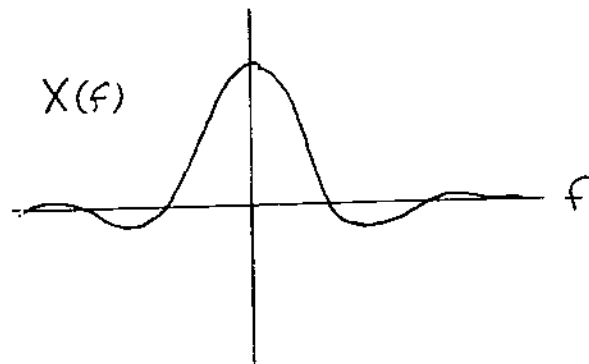
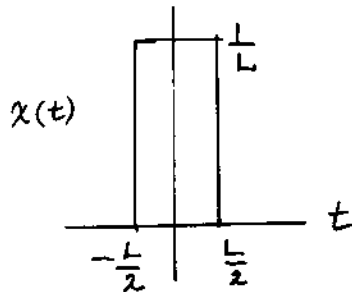
$$\begin{aligned}
x(t) &= \int_0^{\infty} f(t-\xi) g(\xi) d\xi \\
X(i\omega) &= \int_{-\infty}^{\infty} x(t) e^{-i\omega t} dt = \int_{-\infty}^{\infty} \int_0^{\infty} f(t-\xi) g(\xi) d\xi e^{-i\omega t} dt \\
&= \int_{-\infty}^{\infty} \int_0^{\infty} f(t-\xi) e^{-i\omega(t-\xi)} dt g(\xi) e^{-i\omega\xi} d\xi \\
X(i\omega) &= \int_0^{\infty} \left[\int_{-\infty}^{\infty} f(t-\xi) e^{-i\omega(t-\xi)} dt \right] g(\xi) e^{-i\omega\xi} d\xi \\
&\quad \text{Let } (t-\xi) = \tau \quad dt = d\tau \\
&= \int_0^{\infty} \left[\int_{-\infty}^{\infty} f(\tau) e^{-i\omega\tau} d\tau \right] g(\xi) e^{-i\omega\xi} d\xi \\
&= \int_{-\infty}^{\infty} f(\tau) e^{-i\omega\tau} d\tau \int_0^{\infty} g(\xi) e^{-i\omega\xi} d\xi = F(i\omega) H(i\omega) \\
\overline{x^2} &= \int_0^{\infty} \lim_{T \rightarrow \infty} \frac{1}{2\pi T} X(i\omega) X^*(i\omega) d\omega \\
&= \int_0^{\infty} \lim_{T \rightarrow \infty} \frac{1}{2\pi T} F(i\omega) F^*(i\omega) H(i\omega) H^*(i\omega) d\omega \\
&= \int_0^{\infty} S_F(\omega) |H(i\omega)|^2 d\omega
\end{aligned}$$

$$\begin{aligned}
H(i\omega) &= |H(i\omega)| e^{i\phi(\omega)} \\
H^*(i\omega) &= |H(i\omega)| e^{-i\phi(\omega)} \\
\therefore \frac{H(i\omega)}{H^*(i\omega)} &= e^{i2\phi(\omega)}
\end{aligned}$$

13-40Let $X(f) = \text{F.T. of rectangular pulse}$

$$X(f) = \int_{-\frac{L}{2}}^{\frac{L}{2}} \left(\frac{1}{L}\right) e^{-2\pi f t} dt = \left(\frac{1}{L}\right) \frac{e^{-i2\pi f t}}{-i2\pi f} \Bigg|_{-\frac{L}{2}}^{\frac{L}{2}}$$

$$= \frac{1}{L} \frac{1}{\pi f} \left[\frac{e^{i2\pi f \frac{L}{2}} - e^{-i2\pi f \frac{L}{2}}}{2i} \right] = \frac{\sin \pi f L}{\pi f L}$$

13-41

$$\int_{-\infty}^{\infty} |f(t)| dt = \int_{-\infty}^{\infty} u(t) dt = \int_0^{\infty} 1 dt = \infty$$

\therefore Unit step function cannot have a F.T.

13-42

$$S_{FX}(\omega) = \lim_{T \rightarrow \infty} \frac{1}{2\pi T} F^*(i\omega) X(i\omega)$$

$$= \left[\lim_{T \rightarrow \infty} \frac{1}{2\pi T} F^*(i\omega) F(i\omega) \right] H(i\omega) = S_F(i\omega) H(i\omega)$$

$$S_{XF}(\omega) = \lim_{T \rightarrow \infty} \frac{1}{2\pi T} X^* F = \lim_{T \rightarrow \infty} \frac{1}{2\pi T} F^* H^* F = S_F^*(i\omega) H^*(i\omega)$$

$$\frac{S_{FX}}{S_{XF}} = \frac{S_F |H| e^{i\phi}}{S_F |H| e^{-i\phi}} = e^{i2\phi(\omega)}$$

13-42 Cont:

$$\frac{S_F}{S_{XF}} = \frac{S_X}{S_X H^*} \quad \frac{S_{FX}}{S_F} = \frac{S_X H}{S_X} = H$$

$$\frac{S_F}{S_{XF}} = \frac{1}{a-ib} = \frac{(a+ib)}{(a^2+b^2)} = \frac{H(i\omega)}{|H|^2}$$

13-43

$$\frac{d^2 \bar{u}(x,s)}{dx^2} = \left(\frac{s}{c}\right)^2 \bar{u}(x,s)$$

$$\bar{u}(x,s) = C_1 e^{\frac{sx}{c}} + C_2 e^{-\frac{sx}{c}}$$

$$\bar{F}(x,s) = AE \frac{d\bar{u}}{dx} = AE \frac{s}{c} [C_1 e^{\frac{sx}{c}} - C_2 e^{-\frac{sx}{c}}]$$

$$\bar{F}(0,s) = AE \frac{s}{c} [C_1 - C_2]$$

$$\bar{F}(l,s) = 0 = C_1 e^{\frac{sl}{c}} - C_2 e^{-\frac{sl}{c}} \quad \therefore C_1 = C_2 e^{-\frac{2sl}{c}}$$

$$C_2 = \frac{-c \bar{F}(0,s)}{AEs(1 - e^{-\frac{2sl}{c}})}$$

$$\begin{aligned} \bar{u}(x,s) &= \frac{-c \bar{F}(0,s)}{AEs(1 - e^{-\frac{2sl}{c}})} \left[e^{-\frac{2sl}{c}} e^{\frac{sx}{c}} + e^{-\frac{sx}{c}} \right] \\ &= \frac{-c \bar{F}(0,s) e^{-\frac{sl}{c}}}{AEs(1 - e^{-\frac{2sl}{c}})} \left[e^{\frac{s}{c}(x-l)} + e^{-\frac{s}{c}(x-l)} \right] \end{aligned}$$

13-44

$$p(x,t) = F_0 e^{i\omega t} \delta(x)$$

$$\bar{p}(x,s) = \frac{F_0}{s-i\omega} \delta(x), \quad \bar{F}(0,s) = \int_0^l \bar{p}(x,s) dx = \frac{F_0}{s-i\omega}$$

$$\bar{u}(x,s) = \frac{-c F_0 e^{\frac{sl}{c}}}{s(s-i\omega)AE(1 - e^{-\frac{2sl}{c}})} \left[e^{\frac{s}{c}(x-l)} + e^{-\frac{s}{c}(x-l)} \right]$$

$$\therefore u(x,t) = \frac{-c F_0 e^{i\omega t}}{\omega AE \sin \frac{\omega l}{c}} \cos \frac{\omega l}{c} \left(\frac{x}{l} - 1 \right)$$

$$\sigma = E \frac{du}{dx} = \frac{-c F_0 e^{i\omega t}}{\omega A \sin \frac{\omega l}{c}} \cdot \frac{\omega l}{c} \sin \frac{\omega l}{c} \left(\frac{x}{l} - 1 \right) = \frac{-F_0 e^{i\omega t}}{A \sin \frac{\omega l}{c}} \sin \frac{\omega l}{c} \left(\frac{x}{l} - 1 \right)$$

$$u(x, t) = \sum_{n=0}^{\infty} \phi_n(x) q_n(t)$$

$$\sigma(x, t) = E \frac{du}{dx} = E \sum_{n=1}^{\infty} \phi'_n(x) q_n(t) \quad \text{where } \phi' = \frac{d\phi}{dx}$$

$$\begin{aligned} \overline{\sigma(x, t) \sigma(x', t)} &= \lim_{T \rightarrow \infty} \frac{1}{2T} \int_{-T}^T \sigma(x, t) \sigma(x', t) dt \\ &= \frac{1}{2} \int_{-\infty}^{\infty} \lim_{T \rightarrow \infty} \frac{1}{2\pi T} \sigma^*(x, i\omega) \sigma(x', i\omega) d\omega \end{aligned}$$

$$\sigma(x, i\omega) = E \sum_n Q_n(i\omega) \phi'_n(x)$$

$$\overline{\sigma(x, t) \sigma(x', t)} = \frac{E^2}{2} \sum_{n=1}^{\infty} \sum_{k=1}^{\infty} \phi'_n(x) \phi'_k(x') \int_{-\infty}^{\infty} \lim_{T \rightarrow \infty} \frac{1}{2\pi T} Q_n^*(i\omega) Q_k(i\omega) d\omega$$

$$\ddot{q}_n + \omega_n^2 (1 + i\gamma) q_n = \frac{1}{m\ell} \int_0^l \phi(x, t) \delta(x) \phi'_n(x) dx = \frac{1}{m\ell} F(0, t) \phi_n(0)$$

$$Q_n(i\omega) = \frac{F(0, i\omega) \phi_n(0)}{m\ell[(\omega_n^2 - \omega^2) + i\gamma\omega_n^2]} = \frac{F(0, i\omega) \phi_n(0)}{m\ell\omega_n^2 [1 - (\frac{\omega}{\omega_n})^2 + i\gamma]}$$

$$\begin{aligned} &\int_{-\infty}^{\infty} \lim_{T \rightarrow \infty} \frac{1}{2\pi T} \frac{F^*(0, i\omega) F(0, i\omega) \phi_n(0) \phi_k(0)}{m^2 \ell^2 \omega_n^2 \omega_k^2 [1 - (\frac{\omega}{\omega_n})^2 - i\gamma] [1 - (\frac{\omega}{\omega_k})^2 + i\gamma]} d\omega \\ &= \int_{-\infty}^{\infty} \frac{S(i\omega) \phi_n(0) \phi_k(0)}{m^2 \ell^2 \omega_n^2 \omega_k^2 [1 - (\frac{\omega}{\omega_n})^2 - i\gamma] [1 - (\frac{\omega}{\omega_k})^2 + i\gamma]} d\omega \end{aligned}$$

Greatest contribution occurs when $k = n$ \therefore change double \sum to single summation with $k = n$.

13-45 Cont:

$$\overline{\sigma(x,t)\sigma(x',t)} = \frac{E^2}{2} \sum_n \phi_n'(x) \phi_n'(x') \phi_n^2(0) \frac{1}{m^2 l^2 \omega_n^4} \int_{-\infty}^{\infty} \frac{S(i\omega) d\omega}{[1 - (\frac{\omega}{\omega_n})^2]^2 + \gamma^2}$$

$$\overline{\sigma^2(x,t)} = \frac{E^2}{2} \sum_n \phi_n^2(x) \phi_n^2(0) \frac{1}{m^2 l^2 \omega_n^3} S(\omega_n) \frac{\pi}{\gamma}$$

where,

$$\phi_n = \sqrt{2} \cos n\pi \left(\frac{x}{l} - 1\right)$$

$$\omega_n = n\pi \frac{C}{l}$$

$$C = \sqrt{\frac{AE}{m}}$$

$$E = \frac{C^2 m}{A}$$

$$\therefore \overline{\sigma^2(x,t)} \approx \frac{2\pi}{\gamma} \sum \frac{C}{A^2 n\pi l} S(\omega_n) \sin^2 \frac{n\pi x}{l}$$

13-46

Prove $FT[x(t-t_0)] = e^{-i2\pi f t_0} X(f)$

where $X(f) = FT[x(t)]$

From Eq.(13.6-1) $x(t-t_0) = \int_{-\infty}^{\infty} X(f) e^{i2\pi f(t-t_0)} df$
 $= \int_{-\infty}^{\infty} [e^{-i2\pi f t_0} X(f)] e^{i2\pi f t} df$

Comparison with Eq.(13.6-2) shows $e^{-i2\pi f t_0} X(f) = FT[x(t-t_0)]$

13-47

Prove $FT[x(t)*y(t)] = X(f)Y(f)$

$x(t)*y(t) = \int_{-\infty}^{\infty} x(\tau) y(t-\tau) d\tau = \text{convolution of } x(t) \text{ and } y(t)$

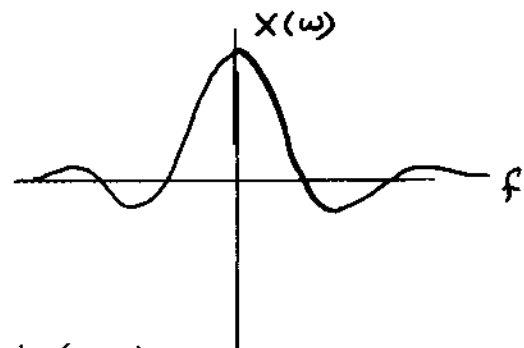
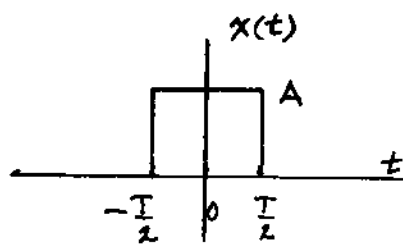
From Eq.(13.6-2) $FT[x(t)*y(t)] = \int_{-\infty}^{\infty} \left[\int_{-\infty}^{\infty} x(\tau) y(t-\tau) d\tau \right] e^{-i2\pi f t} dt$

$= \int_{-\infty}^{\infty} \left[\int_{-\infty}^{\infty} y(t-\tau) e^{-i2\pi f t} dt \right] x(\tau) d\tau$

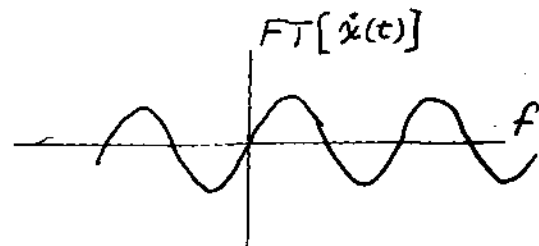
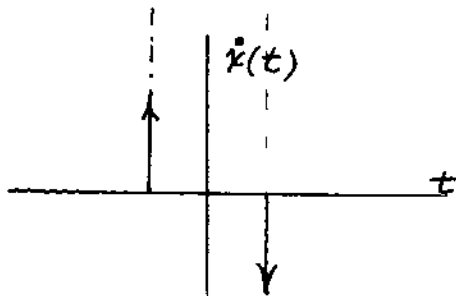
Let $(t-\tau) = \xi$, $t = \xi + \tau$ $dt = d\xi$

$= \int_{-\infty}^{\infty} \left[\int_{-\infty}^{\infty} y(\xi) e^{-i2\pi f \xi} d\xi \right] x(\tau) e^{-i2\pi f \tau} d\tau$

$= \int_{-\infty}^{\infty} x(\tau) e^{-i2\pi f \tau} d\tau \cdot \int_{-\infty}^{\infty} y(\xi) e^{-i2\pi f \xi} d\xi = X(f) Y(f)$



$$X(\omega) = \text{FT}[x(t)] = A T \frac{\sin(\frac{\omega T}{2})}{(\frac{\omega T}{2})}$$



$$\begin{aligned} \text{FT}[\dot{x}(t)] &= i\omega \text{FT}[x(t)] = i\cancel{\omega} \cdot A\cancel{T} \frac{\sin(\frac{\omega T}{2})}{\cancel{\omega T} \cancel{2}} \\ &= i 2A \sin(\frac{\omega T}{2}) = \text{sine wave} \end{aligned}$$

14-1 If $x_1 = \varphi_1(t)$ and $x_2 = \varphi_2(t)$ are solutions of the equation $\ddot{x} + x^3 = 0$, then they will satisfy

$$\ddot{\varphi}_1 + \varphi_1^3 = 0 \quad \text{and} \quad \ddot{\varphi}_2 + \varphi_2^3 = 0$$

Adding, the following is also true

$$(\ddot{\varphi}_1 + \ddot{\varphi}_2) + (\varphi_1^3 + \varphi_2^3) = 0 \quad (a)$$

If we assume $x = \varphi_1 + \varphi_2$ and substitute into the D.E. we would obtain

$$(\ddot{\varphi}_1 + \ddot{\varphi}_2) + (\varphi_1^3 + 3\varphi_1^2\varphi_2 + 3\varphi_1\varphi_2^2 + \varphi_2^3) = 0$$

which does not agree with the correct result (a)

Superposition of solutions are in general not solutions of nonlinear equations.

14-2 $\sum F_x = -2T \sin \theta = m\ddot{x}$

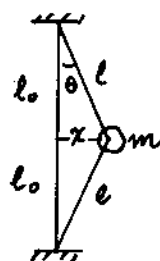
$$\sin \theta = \frac{x}{l} = \frac{x}{\sqrt{l_0^2 + x^2}} \approx \frac{x}{l_0} \left[1 - \frac{1}{2} \left(\frac{x}{l_0} \right)^2 \right]$$

$$T = T_0 + k(l - l_0) = T_0 + k \left[l_0 \left(1 + \frac{x^2}{l_0^2} \right)^{\frac{k}{2}} - l_0 \right]$$

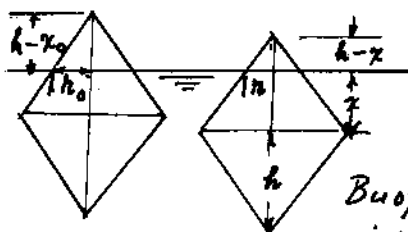
$$\approx T_0 + k \frac{1}{2} \left(\frac{x}{l_0} \right)^2$$

$$\therefore m\ddot{x} + 2 \left[T_0 + \frac{k}{2} \left(\frac{x}{l_0} \right)^2 \right] \frac{x}{\sqrt{l_0^2 + x^2}} = 0$$

$$m\ddot{x} + \frac{2}{l_0} \left[T_0 + \frac{k}{2} \left(\frac{x}{l_0} \right)^2 \right] \left[1 - \frac{1}{2} \left(\frac{x}{l_0} \right)^2 \right] x = 0$$



14-3



$$\text{Vol. of cone} = \frac{1}{3} \pi r^2 (h-x)$$

$$\text{From similar triangles } \frac{r_0}{h-x_0} = \frac{r}{h-x}$$

$$\therefore r = r_0 \left(\frac{h-x}{h-x_0} \right)$$

$$\text{Difference in vol.} = \frac{1}{3} \pi [r^2 (h-x) - r_0^2 (h-x_0)]$$

$$\text{Buoyant force} = \rho \Delta V = \frac{\pi}{3} \rho r_0^2 \left[\left(\frac{h-x}{h-x_0} \right)^2 (h-x) - (h-x_0) \right]$$

= weight of water displaced.

$$\therefore m\ddot{x} = \frac{\pi}{3} \rho r_0^2 \left[\frac{(h-x)^3}{(h-x_0)^2} - (h-x_0) \right]$$

$$= \frac{\pi}{3} \rho \frac{r_0^3}{(h-x_0)^2} \left[(h-x)^3 - (h-x_0)^3 \right]$$

14-4

For $x > x_0$ Eq. of motion is

$$m\ddot{x} + k(x - x_0) = 0 \quad \dots \dots \dots (a)$$

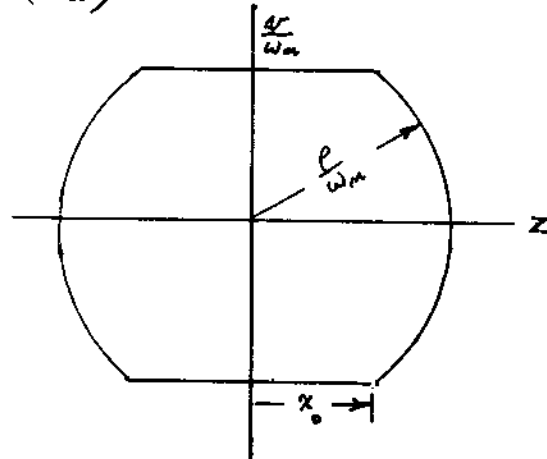
For $x < x_0$ $\ddot{x} = \dot{x} \frac{d\dot{x}}{dx} = 0 \quad \therefore \frac{d\dot{x}}{dx} = 0$ where $v = \dot{x}$

Let $z = (x - x_0)$ in Eq. (a); then

$$\ddot{z} + \omega_m^2 z = 0 \quad \text{or} \quad \dot{z} \frac{d\dot{z}}{dz} + \omega_m^2 z = 0$$

Integrating $\dot{z}^2 + \omega_m^2 z^2 = \left(\frac{p}{\omega_m}\right)^2$ a circle

Phase-plane trajectory
will look like this



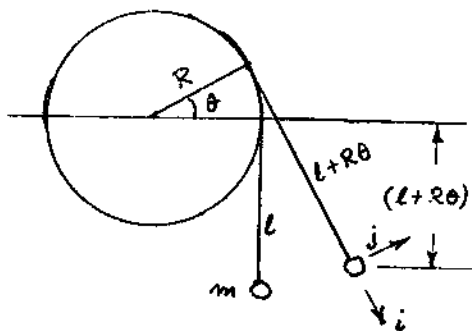
14-5

Since system is conservative

$$\frac{d}{dt}(T + U) = 0 \quad (a)$$

Vel. of m is

$$\vec{v} = (l + R\dot{\theta})\dot{\theta} \vec{j}$$



$$T = \frac{1}{2} m \vec{v} \cdot \vec{v} = \frac{1}{2} m \dot{\theta}^2 (l + R\dot{\theta})^2$$

$$U = mg[l - (l + R\dot{\theta})\cos\theta + R\sin\theta]$$

Subst. into (a)

$$\dot{\theta} \left\{ \ddot{\theta} (l + R\dot{\theta})^2 + \dot{\theta} (l + R\dot{\theta}) R + g(l + R\dot{\theta}) \sin\theta - R\dot{\theta} \cos\theta + R\dot{\theta} \sin\theta \right\} = 0$$

$$\ddot{\theta} + \frac{R\dot{\theta}^2}{l + R\dot{\theta}} + \frac{g}{l + R\dot{\theta}} \sin\theta = 0$$

14-6

$$\ddot{x} + \omega_m^2 x = 0$$

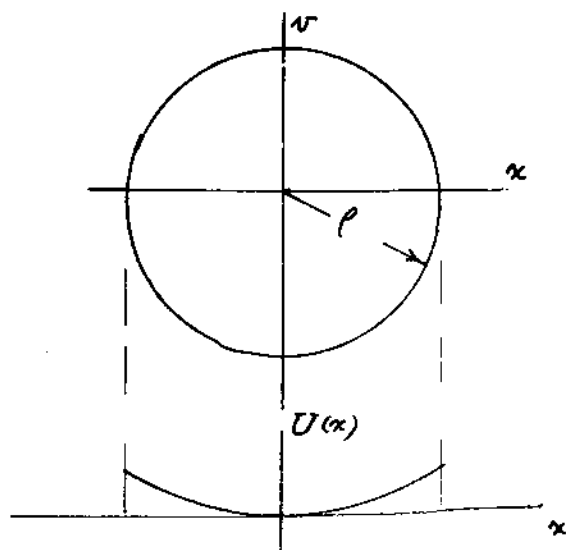
$$\ddot{x} = \dot{x} \frac{d\dot{x}}{dx}, \quad \text{Let } v = \frac{\dot{x}}{\omega_m}$$

then above eq. becomes

$$\frac{dv}{dx} = -\frac{x}{v} \quad \text{or} \quad v^2 + x^2 = \rho^2$$

phase-plane traj. is a circle

$$U = \frac{1}{2} k x^2 \quad \text{a parabola}$$



14-7

$$\gamma = 4 \int_0^{x_{\max}} \frac{dx}{\sqrt{2[E - U(x)]}}$$

$$U(x) = \frac{1}{2} \frac{k}{m} x^2 \quad \text{per unit mass}$$

$$E = \frac{1}{2} \dot{x}^2 + \frac{1}{2} \frac{k}{m} x^2$$

$$\dot{x} = 0 \quad \text{when } x = x_{\max} \quad \therefore x_{\max} = \sqrt{\frac{2Em}{k}}$$

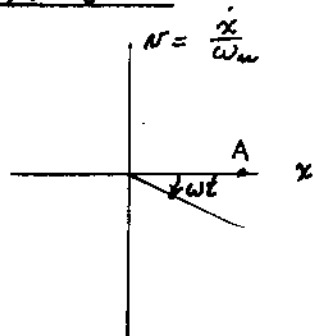
$$\gamma = 4 \int_0^{\sqrt{\frac{2Em}{k}}} \frac{dx}{\sqrt{2\left[E - \frac{1}{2} \frac{k}{m} x^2\right]}}$$

$$\text{but } \frac{k}{m} x^2 = \omega_m^2 x^2$$

$$C^2 = 2E$$

$$\gamma = \frac{4}{\omega_m} \int_0^{u = \omega_m x_{\max} = C} \frac{du}{\sqrt{C^2 - u^2}} = \frac{4}{\omega_m} \sin^{-1}\left(\frac{u}{C}\right) \Bigg|_0^C = \frac{4}{\omega_m} \frac{\pi}{2} = \frac{2\pi}{\omega_m}$$

14-8



$$x(0) = A$$

$$\dot{x}(0) = 0$$

$$\text{Let } y = \dot{x}$$

$$\dot{y} = -\omega_m^2 x$$

$$V = \sqrt{\dot{x}^2 + \dot{y}^2} = \sqrt{y^2 + \omega_m^4 x^2} = 0$$

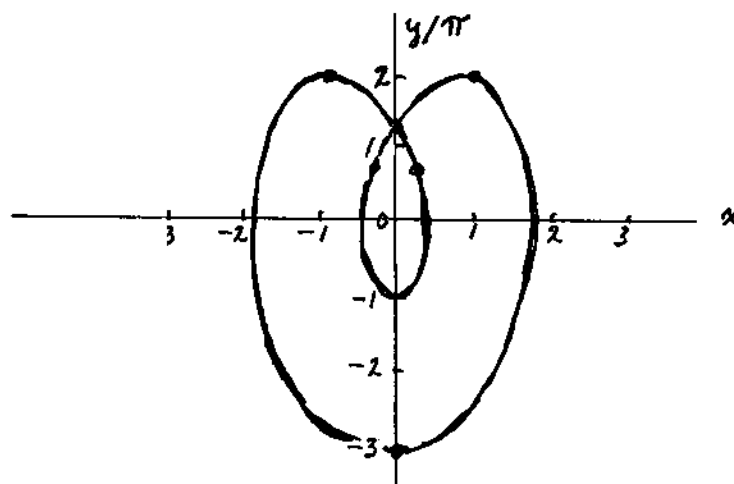
$$V = 0 \quad \text{only if } x = y = 0$$

14-9

$$x = \cos \pi t + \sin 2\pi t$$

$$y = \dot{x} = -\pi \sin \pi t + 2\pi \cos 2\pi t$$

t	x	y
0	1	2π
.25	-.3	.7π
.50	0	-π
.75	.3	.7π
1.0	-1	2π
1.25	-1.7	-.7π
1.5	0	-3π
1.75	1.7	-.7π
2.0	1	2π



14-10

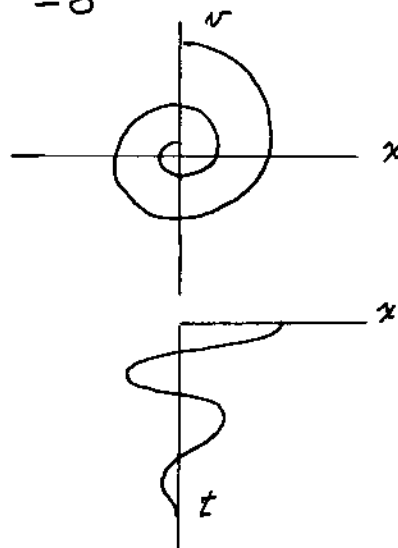
$$\ddot{x} + 2.5 \omega_m \dot{x} + \omega_m^2 x = 0$$

$$\dot{x} \frac{d\dot{x}}{dx} = -2.5 \omega_m \dot{x} - \omega_m^2 x$$

$$\frac{1}{\omega_m} \frac{d\dot{x}}{dx} = -2.5 - \omega_m \frac{x}{\dot{x}}$$

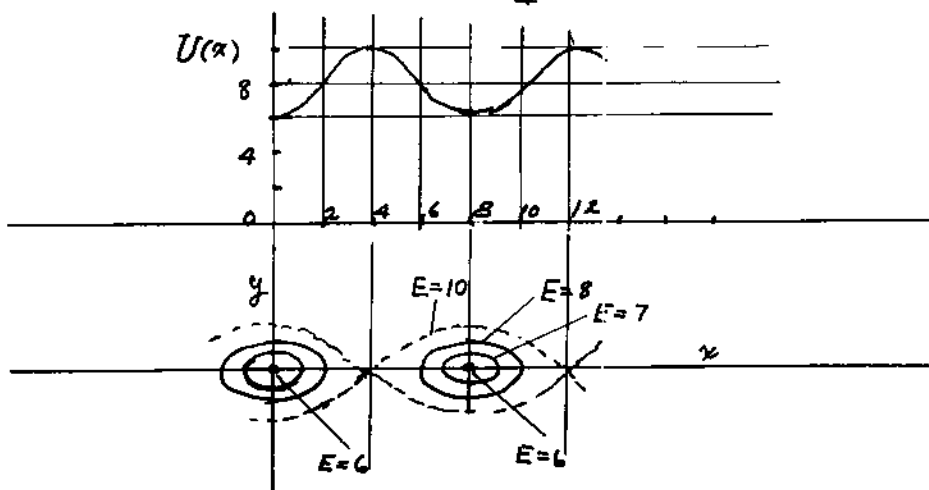
$$\text{Let } v = \frac{\dot{x}}{\omega_m}$$

$$\frac{dv}{dx} = -2.5 - \frac{x}{v} \quad \text{Trajectory is a spiral.}$$



14-12

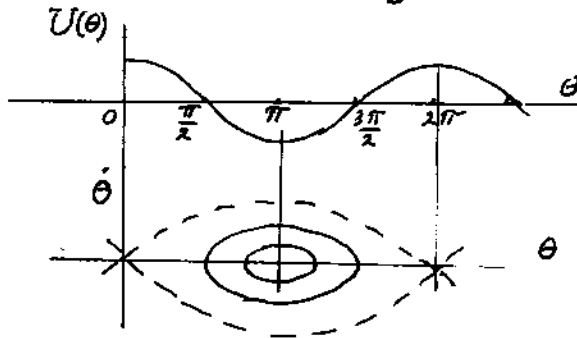
$$U = 8 - 2 \cos \frac{\pi x}{4}$$



see next page for Problem 11

14-11

$$U = \frac{g}{L} \cos \theta$$



The origin of phase plane is shifted to π , as compared to FIG 14.4-2

$\theta = 0$ and 2π are unstable points

14-13

$$\frac{dy}{dx} = \frac{2x + 2y}{5x - y} = \frac{P}{Q}$$

singular points are $\frac{P}{Q} = \frac{0}{0}$

$$\begin{aligned} 2x + 2y &= 0 \\ 5x - y &= 0 \\ \hline 12x &= 0 \end{aligned}$$

$$\therefore x = 0, \quad y = 0$$

$$\begin{Bmatrix} \dot{u} \\ \dot{v} \end{Bmatrix} = \begin{bmatrix} 5 & -1 \\ 2 & 2 \end{bmatrix} \begin{Bmatrix} u \\ v \end{Bmatrix} \quad \text{Eq. 14.3-8}$$

$(a + e) > 0 \quad \therefore$ system is unstable and aperiodic

$$\begin{aligned} \lambda_1 &= 3.0 & u &= e^{3t} \\ \lambda_2 &= 4.0 & v &= e^{4t} \end{aligned}$$

14-14

$$\begin{Bmatrix} \dot{x} \\ \dot{y} \end{Bmatrix} = \begin{bmatrix} 5 & -1 \\ 2 & 2 \end{bmatrix} \begin{Bmatrix} x \\ y \end{Bmatrix}$$

$$\begin{vmatrix} (5-\lambda) & -1 \\ 2 & (2-\lambda) \end{vmatrix} = 0$$

$$\text{gives } \lambda = \begin{Bmatrix} 3 \\ 4 \end{Bmatrix}$$

subst λ into eq.

$$(5-\lambda)x = y \quad \text{gives} \quad \begin{aligned} x^{(1)} &= 0.5 y^{(1)} \\ x^{(2)} &= 1.0 y^{(2)} \end{aligned}$$

$$P = \begin{bmatrix} 0.5 & 1 \\ 1 & 1 \end{bmatrix}$$

Transformation to decouple is

$$\begin{Bmatrix} x \\ y \end{Bmatrix} = \begin{bmatrix} 0.5 & 1 \\ 1 & 1 \end{bmatrix} \begin{Bmatrix} \xi \\ \eta \end{Bmatrix}$$

$$\begin{bmatrix} 0.5 & 1 \\ 1 & 1 \end{bmatrix} \begin{Bmatrix} \dot{\xi} \\ \dot{\eta} \end{Bmatrix} = \begin{bmatrix} 5 & -1 \\ 2 & 2 \end{bmatrix} \begin{bmatrix} 0.5 & 1 \\ 1 & 1 \end{bmatrix} \begin{Bmatrix} \xi \\ \eta \end{Bmatrix}$$

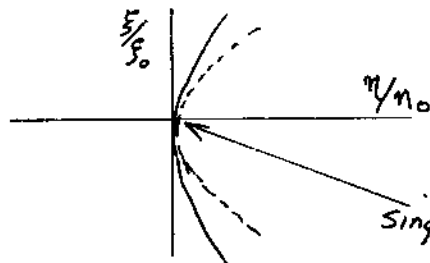
$$\begin{Bmatrix} \dot{\xi} \\ \dot{\eta} \end{Bmatrix} = \begin{bmatrix} 0.5 & 1 \\ 1 & 1 \end{bmatrix}^{-1} \begin{bmatrix} 5 & -1 \\ 2 & 2 \end{bmatrix} \begin{bmatrix} 0.5 & 1 \\ 1 & 1 \end{bmatrix} \begin{Bmatrix} \xi \\ \eta \end{Bmatrix} = \begin{bmatrix} 3 & 0 \\ 0 & 4 \end{bmatrix} \begin{Bmatrix} \xi \\ \eta \end{Bmatrix}$$

$$\therefore \dot{\xi} = 3\xi \quad \text{and} \quad \dot{\eta} = 4\eta \quad \text{decoupled eqs.}$$

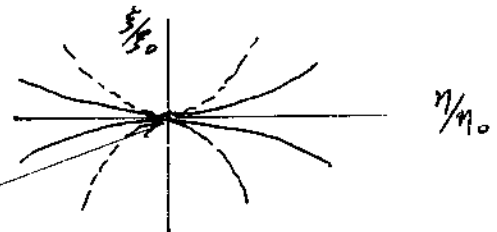
1. - 15 Uncoupled eqs. $\begin{cases} \dot{\xi} = \lambda_1 \xi \\ \dot{\eta} = \lambda_2 \eta \end{cases}$ can be written as

$$\frac{d\xi}{d\eta} = \left(\frac{\lambda_1}{\lambda_2} \right) \frac{\xi}{\eta} \quad \text{Integrating} \quad \xi = \xi_0 \left(\frac{\eta}{\eta_0} \right)^{\frac{\lambda_1}{\lambda_2}}$$

For $\frac{\lambda_1}{\lambda_2} = .5$, the traj. are tangent to ξ as shown



If $\frac{\lambda_1}{\lambda_2} = 2$, the traj. are tangent to η



Singular pts

1 - 16 $\frac{d\xi}{d\eta} = 2 \frac{\xi}{\eta} \quad \therefore \xi = \xi_0 \left(\frac{\eta}{\eta_0} \right)^2$

$$\begin{Bmatrix} u \\ v \end{Bmatrix} = \begin{bmatrix} p & \end{bmatrix} \begin{Bmatrix} \xi \\ \eta \end{Bmatrix} = \begin{bmatrix} p & \end{bmatrix} \begin{Bmatrix} \xi_0 \left(\frac{\eta}{\eta_0} \right)^2 \\ \eta \end{Bmatrix}$$

\therefore Eq for u & v are of the form

$$\left. \begin{aligned} u &= A \eta^2 + B \eta = x - x_s \\ v &= C \eta^2 + D \eta = y - y_s \end{aligned} \right\} \begin{array}{l} \text{only a linear} \\ \text{shift of origin} \end{array}$$

Need original eq.

$$\begin{Bmatrix} \dot{u} \\ \dot{v} \end{Bmatrix} = \begin{bmatrix} a & b \\ c & e \end{bmatrix} \begin{Bmatrix} u \\ v \end{Bmatrix} \quad \text{corresponding to } \frac{\lambda_1}{\lambda_2} = 2$$

a, b, c, e must be known before plotting.

Use Eq. 12.3-9

$$\begin{Bmatrix} u \\ v \end{Bmatrix} = \begin{bmatrix} u_1 & u_2 \\ v_1 & v_2 \end{bmatrix} \begin{Bmatrix} \xi \\ \eta \end{Bmatrix} = P \begin{Bmatrix} \xi \\ \eta \end{Bmatrix}$$

From Eq. 14.3-8

$$\begin{vmatrix} (a-\lambda) & b \\ c & (e-\lambda) \end{vmatrix} = 0 \quad \therefore$$

$$\frac{u_1}{v_1} = \frac{-b}{a-\lambda_1}$$

$$\frac{u_2}{v_2} = \frac{-b}{a-\lambda_2}$$

14-16 Cont. Since only relative values of u, v are essential, Let $v_1 = v_2 = 1.0$ and $u_1 = \frac{-b}{a-\lambda_1}$, $u_2 = \frac{-b}{a-\lambda_2}$.
Using Eq. 14.3-8 we have

$$P \begin{Bmatrix} \dot{\xi} \\ \dot{\eta} \end{Bmatrix} = \begin{bmatrix} a & b \\ c & e \end{bmatrix} P \begin{Bmatrix} \xi \\ \eta \end{Bmatrix}$$

$$\begin{Bmatrix} \dot{\xi} \\ \dot{\eta} \end{Bmatrix} = P^{-1} \begin{bmatrix} a & b \\ c & e \end{bmatrix} P \begin{Bmatrix} \xi \\ \eta \end{Bmatrix} = [\Lambda] \begin{Bmatrix} \xi \\ \eta \end{Bmatrix}$$

$$\therefore P^{-1} \begin{bmatrix} a & b \\ c & e \end{bmatrix} P = \begin{bmatrix} \lambda_1 & 0 \\ 0 & \lambda_2 \end{bmatrix} \text{ where } P = \begin{bmatrix} \frac{-b}{a-\lambda_1} & \frac{-b}{a-\lambda_2} \\ 1 & 1 \end{bmatrix}$$

Equating the two elements on the two sides of the above eq.

$$\left. \begin{aligned} (au_1 + b) - (cu_1 + e) &= 0 \\ [-u_1(au_1 + b) + u_1(cu_1 + e)] &= 0 \\ \frac{1}{(u_1 v_2 - u_2 v_1)} [(au_1 + b) - (cu_1 + e)] &= \lambda_1 \\ \frac{1}{(u_1 v_2 - u_2 v_1)} [-u_1(au_1 + b) + u_1(cu_1 + e)] &= \lambda_2 \end{aligned} \right\} \begin{aligned} &\text{These 4 eqs. can be solved} \\ &\text{for } a, b, c, e, \text{ then subst.} \\ &\text{into Eq. 14.3-9} \\ &\text{Other alternative is to} \\ &\text{solve for } u \text{ \& } v \text{ from} \\ &\text{Eq. 14.3-12 for chosen values} \\ &\text{of } t. \end{aligned}$$

14-17 From given eq.

$$\begin{Bmatrix} \dot{v} \\ \dot{u} \end{Bmatrix} = \begin{bmatrix} \alpha & \beta \\ -\beta & \alpha \end{bmatrix} \begin{Bmatrix} v \\ u \end{Bmatrix} \quad \begin{vmatrix} (\alpha - \lambda) & -\beta \\ -\beta & (\alpha - \lambda) \end{vmatrix} = 0$$

$$\therefore \lambda^2 - 2\alpha\lambda + \alpha^2 + \beta^2 = 0 \quad \text{and} \quad \lambda = -\alpha \pm i\beta$$

14-18 $u = \rho \cos \theta$ $v = \rho \sin \theta$

$$du = d\rho \cos \theta - \rho \sin \theta d\theta \quad dv = d\rho \sin \theta + \rho \cos \theta d\theta$$

Subst. u & v in Eq. for Prob. 14-17

$$\frac{dv}{du} = \frac{\beta \rho \cos \theta + \alpha \rho \sin \theta}{\alpha \rho \cos \theta - \beta \rho \sin \theta} = \frac{d\rho \sin \theta + \rho \cos \theta d\theta}{d\rho \cos \theta - \rho \sin \theta d\theta}$$

$$\begin{aligned} (d\rho \sin \theta + \rho \cos \theta d\theta)(\alpha \cos \theta - \beta \sin \theta) &= (d\rho \cos \theta - \rho \sin \theta d\theta)(\beta \cos \theta + \alpha \sin \theta) \\ d\rho [\alpha \cos \theta \sin \theta - \beta (\sin^2 \theta + \cos^2 \theta) - \alpha \sin \theta \cos \theta] &= -d\theta [\alpha \cos^2 \theta - \beta \sin \theta \cos \theta + \alpha \sin \theta \cos \theta + \beta \sin^2 \theta] \end{aligned}$$

$$\therefore \frac{d\rho}{\rho} = \frac{\alpha}{\beta} d\theta \quad \rho = e^{\frac{\alpha}{\beta} \theta}$$

14-19

Eq. in x, y plane is $xy = \pm C$

node point at origin is unstable

$$x dy + y dx = 0 \quad \therefore \frac{dy}{dx} = -\frac{y}{x}$$

$$\begin{Bmatrix} \dot{y} \\ \dot{x} \end{Bmatrix} = \begin{bmatrix} -1 & 0 \\ 0 & 1 \end{bmatrix} \begin{Bmatrix} y \\ x \end{Bmatrix} \quad \begin{vmatrix} -(1+\lambda) & 0 \\ 0 & (1-\lambda) \end{vmatrix} = 0$$

$$(1-\lambda)(1+\lambda) = 0 \quad \lambda = \pm 1 \quad \begin{aligned} \xi &= \xi_0 e^t \\ \eta &= \eta_0 e^{-t} \end{aligned}$$

$$\frac{d\xi}{d\eta} = \frac{\lambda_1 \xi}{\lambda_2 \eta} = -\frac{\xi}{\eta} \quad \text{or} \quad \eta d\xi + \xi d\eta = 0$$

$\therefore \xi, \eta$ plot is $\xi\eta = \pm \text{const.} \quad \therefore \text{same}$

14-20

The form of the eq. is

$$\frac{dv}{du} = \frac{v+u}{u}$$

$$\begin{Bmatrix} \dot{v} \\ \dot{u} \end{Bmatrix} = \begin{bmatrix} 1 & 1 \\ 0 & 1 \end{bmatrix} \begin{Bmatrix} v \\ u \end{Bmatrix}$$

Characteristic eq.

$$\begin{vmatrix} (1-\lambda) & 1 \\ 0 & (1-\lambda) \end{vmatrix} = 0 \quad \text{Leads to two equal roots } \lambda = 1$$

Transformation eq. 14.3-9 cannot be applied

14-21

Differentiate the eq. $x^2 + 2xy + 3y^2 = C$

$$2x dx + 2x dy + 2y dx + 6y dy = 0$$

$$(2x + 6y) dy = -(2x + 2y) dx \quad \therefore \frac{dy}{dx} = \frac{-x - 3y}{x + 3y}$$

14-22

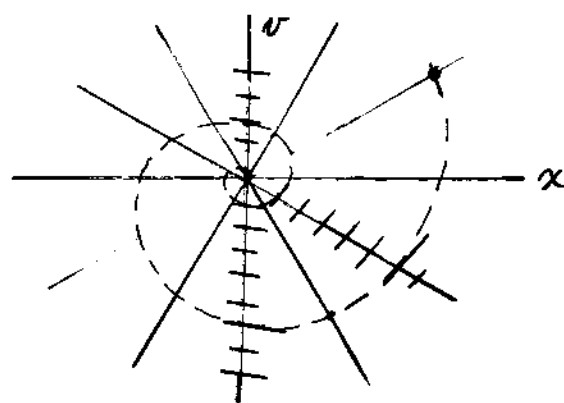
Let $v = \dot{x}$ then the eq. $\ddot{x} + 2\zeta\omega_n \dot{x} + \omega_n^2 x = 0$ becomes $\dot{v} + 2\zeta\omega_n v + \omega_n^2 x = 0$, $v \frac{dv}{dx} + 2\zeta\omega_n v + \omega_n^2 x = 0$

Let $\frac{dv}{dx} = \text{constant for isocline} = C_1$

$$v(C_1 + 2\zeta\omega_n) + \omega_n^2 x = 0$$

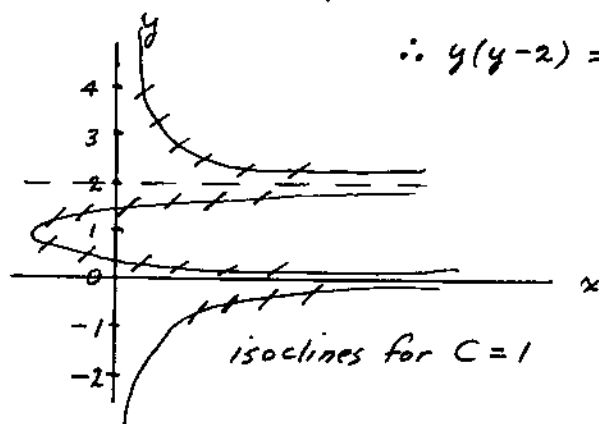
= straight line through origin

As $t \rightarrow \infty$ the points on the isoclines move towards the origin \therefore stable



14-23

$$\frac{dy}{dx} = xy(y-2) = \text{constant for isocline}$$



$$\therefore y(y-2) = \frac{C}{x}$$

y	$y(y-2)$	$x \text{ for } C=1$
-3	15	.066
-2	8	.125
-1	3	.333
0	0	∞
1	-1	-1
2	0	∞
3	3	.333
4	8	.125

14-24

$$y \frac{dy}{dx} + \omega_m^2 x + \mu x^3 = 0$$

$$\text{integrating } y^2 + \omega_m^2 x^2 + \frac{1}{2} \mu x^4 = 2E$$

$$\text{or } 2T + 2U = 2E$$

$$\text{Since } y = \frac{dx}{dt} \quad dt = \frac{dx}{y} = \frac{dx}{\sqrt{2E - 2U}}$$

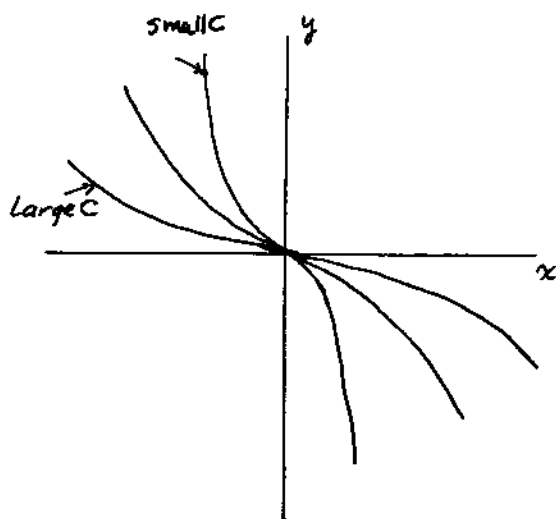
$$\text{For } \frac{1}{4} \text{ period} \quad \frac{T}{4} = \int_0^A \frac{dx}{\sqrt{2(E-U)}}$$

14-25

From Prob. 14-24

$$\frac{dy}{dx} = \frac{-\omega_m^2 x - \mu x^3}{y} = C$$

$$\text{Let } \frac{\omega_m^2}{C} = 4 + \frac{\mu}{C} = 2$$



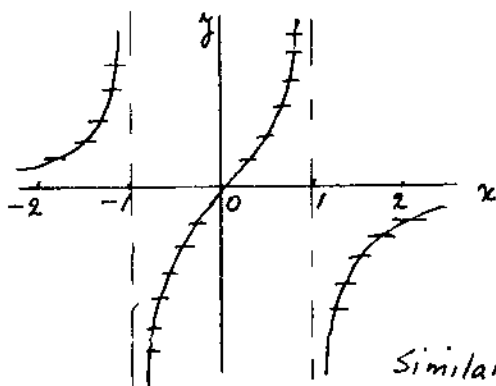
x	$4+2x^2$	$y = -x(4+2x^2)$
-1	6	6
0	4	0
1	6	-6
2	12	-24
3	22	-66

14-26 $\ddot{x} - \mu \dot{x}(1-x^2) + x = 0$

$y \frac{dy}{dx} - \mu y(1-x^2) + x = 0$

$\frac{dy}{dx} = \frac{\mu y(1-x^2) - x}{y} = \mu(1-x^2) - \frac{x}{y} = \text{const}$

For $\frac{dy}{dx} = C = 0$ and $\mu = 2$



x	y
0	0
±.2	±.104
±.4	±.139
±.6	±.45
±.8	±1.11
±.9	±2.37
±1.0	±∞
±2	∓.333
±4	∓.13

Similarly solve for $C = \pm 1$ and other values

14-27 Let $\omega_n^2 = \frac{k}{m}$ $\tau = \omega_n t$ $y = \frac{dx}{d\tau}$

Then $\dot{x} = \omega_n y$ $\ddot{x} = \omega_n^2 y \frac{dy}{dx}$

the eq $\ddot{x} + \frac{c}{m} \dot{x} + \frac{k}{m} x + \frac{\mu}{m} x^3 = 0$ becomes

$y \frac{dy}{dx} + \frac{c}{m} y + x + \frac{\mu}{\omega_n^2 m} x^3 = 0$

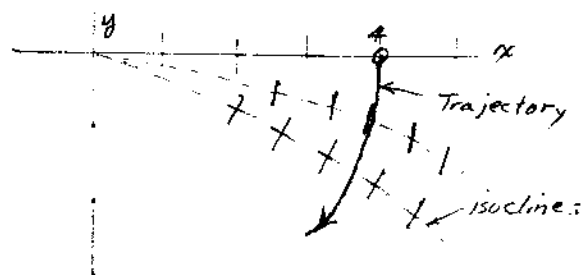
$\frac{dy}{dx} = \frac{-[x + (\frac{c}{m} y + \frac{\mu}{\omega_n^2 m} x^3)]}{y} = \alpha$

14-28 Above eq. can be written as given: $\omega_n^2 = 25$, $\frac{\mu}{m} = 5$

$(\alpha + \frac{c}{m}) y = -(x + \frac{\mu}{\omega_n^2 m} x^3)$ where $\alpha = \frac{dy}{dx}$ $\frac{c}{m} = 2.0$

$\therefore (\alpha + 2) y = -(x + 0.2 x^3)$ Assign different values for α and plot the field of isoclines. From prob. 14-27 $\dot{x}(0) = \omega_n y(0) = 0$

\therefore Start with $x(0) = 4$, $\dot{x}(0) = 0 \therefore y(0) = 0$ and fill in trajectory



All isoclines are vertical along x axis. The right side of eq. is straight line + cubic which must be \div by $(\alpha + 2)$

14-29

$$\ddot{\theta} + \frac{g}{l} \sin \theta = 0$$

Let $x = \theta$, $\omega_0^2 = g/l$

$$y = \frac{\dot{\theta}}{\omega_0} = \frac{\dot{x}}{\omega_0} = \frac{dx}{d\tau}$$

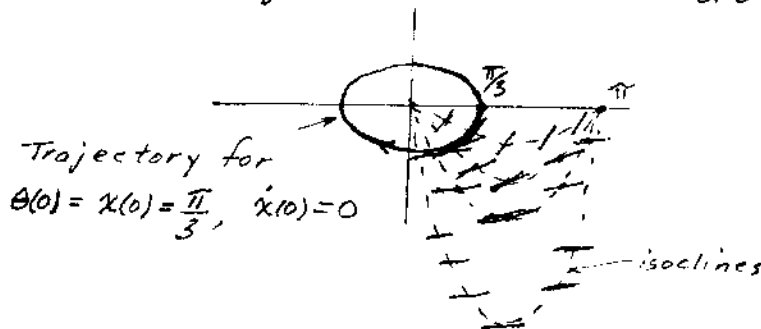
$$\therefore \omega_0^2 y \frac{dy}{dx} + \frac{g}{l} \sin x = 0$$

with $\alpha = \frac{dy}{dx} = \text{constant}$

$$\alpha y = - \left(\frac{g}{l\omega_0^2} \right) \sin x \quad \text{or} \quad y = - \frac{g}{\alpha l\omega_0^2} \sin x = \text{eq. for isocline with } \alpha = \text{const.}$$

Since $\frac{g}{l\omega_0^2} = 1$

$$y = -\frac{1}{\alpha} \sin x = \text{isocline}$$



Trajectory for
 $\theta(0) = x(0) = \frac{\pi}{3}$, $\dot{x}(0) = 0$

14-30

$$\ddot{\theta} + \frac{g}{l} \sin \theta = 0$$

$$\dot{\theta} \frac{d\dot{\theta}}{d\theta} + \frac{g}{l} \sin \theta = 0$$

Integrate $\frac{\dot{\theta}^2}{2} - \frac{g}{l} \cos \theta = E$ $\therefore U = -\frac{g}{l} \cos \theta$

At $t=0$ $\theta = 60^\circ$ $\dot{\theta} = 0$ $\therefore 0 - \frac{g}{l} \cos 60^\circ = E$

Since $\dot{\theta} = \frac{d\theta}{dt}$ $dt = \frac{d\theta}{\dot{\theta}}$ $E = -\frac{g}{2l} = -\frac{g}{l} \cos \theta_0$

$$dt = \frac{d\theta}{\sqrt{2E + 2\frac{g}{l} \cos \theta}}$$

$$t = \int \frac{d\theta}{\sqrt{2(E + \frac{g}{l} \cos \theta)}}$$

$$t = \sqrt{\frac{l}{g}} \int \frac{d\theta}{\sqrt{2(\cos \theta - \cos \theta_0)}}$$

$$\cos \theta = 1 - 2 \sin^2 \frac{\theta}{2}$$

$$\cos \theta - \cos \theta_0 = 2 \left(\sin^2 \frac{\theta_0}{2} - \sin^2 \frac{\theta}{2} \right)$$

Let $\sin \frac{\theta}{2} = \sin \frac{\theta_0}{2} \sin \phi = k \sin \phi$ (i)

then $\cos \theta - \cos \theta_0 = 2 \sin^2 \frac{\theta_0}{2} (1 - \sin^2 \phi) = 2 \sin^2 \frac{\theta_0}{2} \cos^2 \phi$ (ii)

Diff (i) $\frac{1}{2} \cos \frac{\theta}{2} d\theta = k \cos \phi d\phi$ $\therefore d\theta = \frac{2k \cos \phi d\phi}{\cos \frac{\theta}{2}}$ (iii)

Subst (ii) & (iii) into t

$$t = \sqrt{\frac{l}{g}} \int \frac{2k \cos \phi d\phi}{\cos \frac{\theta}{2} \cdot 2 \sin^2 \frac{\theta_0}{2} \cos^2 \phi} = \sqrt{\frac{l}{g}} \int \frac{d\phi}{\cos \frac{\theta}{2}}$$

14-30 Cont:

$$t = \sqrt{\frac{l}{g}} \int \frac{d\phi}{\sqrt{1 - \sin^2 \frac{\theta}{2}}} = \sqrt{\frac{l}{g}} \int \frac{d\phi}{\sqrt{1 - k^2 \sin^2 \phi}}$$

$$\therefore \text{period } T = 4 \sqrt{\frac{l}{g}} \int_0^{\pi/2} \frac{d\phi}{\sqrt{1 - k^2 \sin^2 \phi}} \quad k = \sin \frac{\theta_0}{2}$$

Since $\sin \phi = \frac{\sin \frac{\theta}{2}}{\sin \frac{\theta_0}{2}}$, when $\theta = 0$, $\phi = 0$
 when $\theta = \theta_0$, $\phi = \frac{\pi}{2} = \frac{1}{4} \text{ cycle}$

14-31 $\ddot{x} + \omega_n^2 x + C \operatorname{sgn}(\dot{x}) = 0 \quad T = \omega_n t$

$$\dot{x} \frac{d\dot{x}}{dx} + \omega_n^2 x + C \operatorname{sgn}(\dot{x}) = 0 \quad \text{Let } \frac{dx}{dT} = y = \dot{x}$$

$$\omega_n^2 y \frac{dy}{dx} + \omega_n^2 x + C \operatorname{sgn}(y) = 0$$

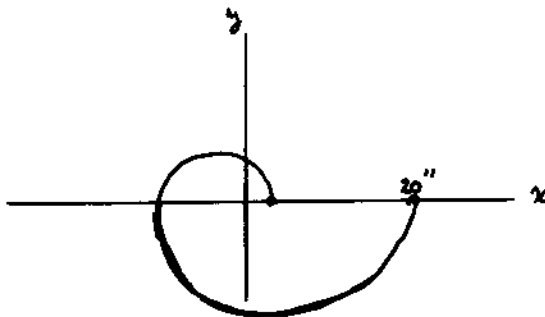
$$\frac{dy}{dx} = - \frac{\frac{1}{\omega_n} C \operatorname{sgn}(y) + x}{y} = - \frac{f(y) + x}{y}$$

where $f(y) = \frac{1}{\omega_n} C \operatorname{sgn}(y)$

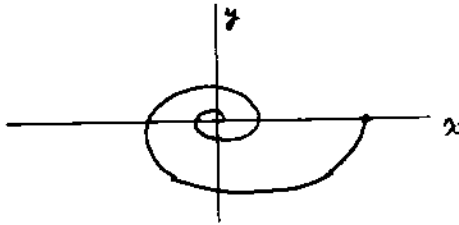
14-32 Initial values $x(0) = 20^\circ \quad y(0) = 0$

$$\omega_n = \sqrt{\frac{3.60}{.10}} = 6, \quad \mu = 0.20$$

$$\frac{\mu g}{\omega_n^2} = \frac{.20 \times 386}{36} = 2.145 \text{ in}$$



14-33 For the undamped pendulum the trajectory is an ellipse, For the damped pendulum the curve is inside of the ellipse, as shown



14-34 $\ddot{\theta} + \omega_m^2 \sin \theta = 0$ $\sin \theta \approx \theta - \frac{\theta^3}{6}$

$$\ddot{\theta} + \omega_m^2 \left(\theta - \frac{\theta^3}{6} \right) = 0$$

Let $\theta = \theta_0 + \mu \theta_1$ and $\omega^2 = \omega_m^2 + \mu \alpha$

From Eq. 12.6-9

$$\omega^2 = \omega_m^2 + \frac{3}{4} \mu A^2 = \omega_m^2 \left[1 + \frac{3}{4} \times \frac{1}{6} \theta_0^2 \right]$$

$$\therefore \omega = \omega_m \sqrt{1 + \frac{1}{8} \theta_0^2} \approx \omega_m \left(1 + \frac{1}{16} \theta_0^2 \right) \quad \omega_m = \sqrt{\frac{g}{L}}$$

14-35 From Prob 14-34

$$\omega = \frac{2\pi}{T} \approx \frac{2\pi}{T_m} \left(1 + \frac{1}{16} \theta_0^2 \right) \therefore T \approx T_m \left(\frac{1}{1 + \frac{1}{16} \theta_0^2} \right)$$

14-36 $\ddot{x} + 0.15 \dot{x} + 10x + x^3 = 5 \cos(\omega t + \phi)$

$$\omega_m^2 = 10, \quad C = .15, \quad \mu = 1.0, \quad F = 5$$

Eq. 14.6-11 becomes

$$25 = \left[(10 - \omega^2)A + \frac{3}{4} A^3 \right]^2 + \left[.15 \omega A \right]^2$$

Rearrange to

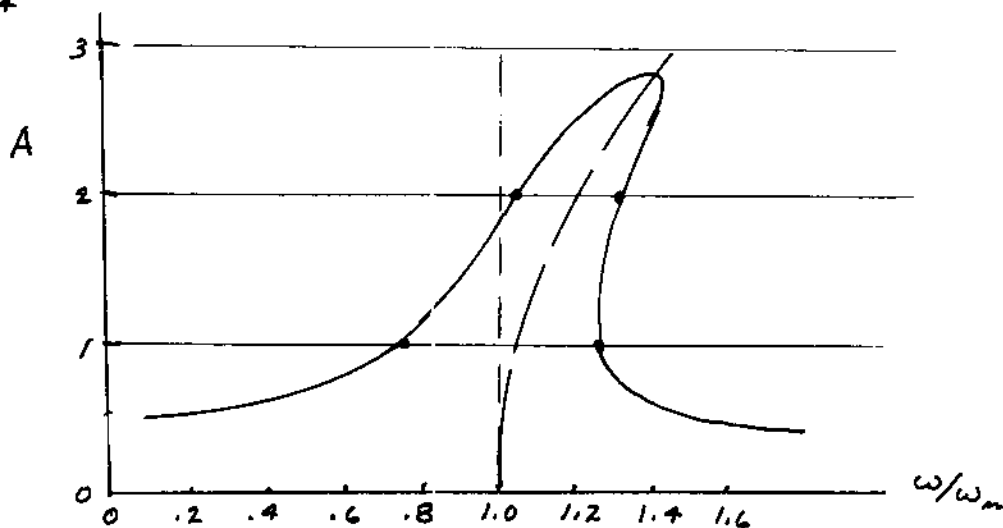
$$\omega^4 - \left(20 + \frac{3}{2} A^2 - .0225 \right) \omega^2 + \left(100 + 15 A^2 + \frac{9}{16} A^4 - \frac{25}{A^2} \right) = 0$$

$$\omega^4 - b \omega^2 + c = 0$$

14-36 Cont.:

A	b	C	ω^2	ω/ω_m
0	19.98	$-\infty$	∞	∞
1	21.48	90.5	$\begin{cases} 5.7 \\ 15.78 \end{cases}$	$\begin{cases} .755 \\ 1.255 \end{cases}$
2	25.98	162.7	$\begin{cases} 10.69 \\ 15.29 \end{cases}$	$\begin{cases} 1.036 \\ 1.24 \end{cases}$
3	32.48	272.7	complex	this indicates the peak to be below $A=3$
$\approx .5$				0

$$\sqrt{\frac{b^2}{4} - C} = 0 \quad \text{gives } A \approx 2.9$$



14-37

$$\frac{\omega/\omega_m}{NS} \quad \varphi_1 \quad \& \quad \varphi_2 \quad \begin{matrix} \varphi_1 = \text{branch 1} \\ \varphi_2 = \text{'' 2} \end{matrix}$$

$$\tan \phi = \frac{B_0}{A_0} = \frac{C\omega}{(\omega_m^2 - \omega^2) + \frac{3}{4}\mu A^2} \quad \text{Eq. 14.6-10}$$

$$= \frac{.15\omega}{(10 - \omega^2) + \frac{3}{4}A^2}$$

Subst. ω & A from Prob. 14-36

$$(\tan \phi)_{A=1} = \frac{.15 \sqrt{5.7}}{(10 - 5.7) + .75} = \frac{.358}{5.05} = .071, \quad \phi = 4^\circ 4'$$

14-37 Cont.

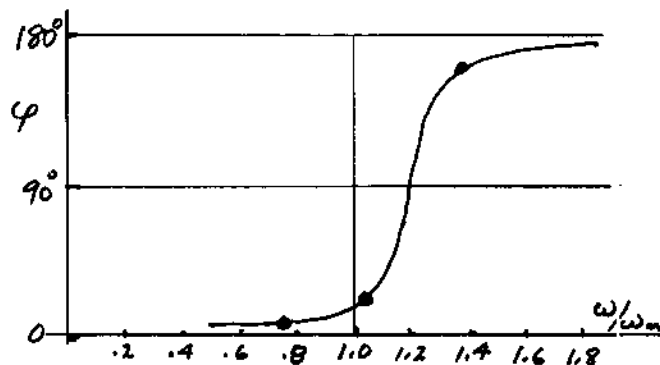
$$(\tan \phi)_{A=1} = \frac{.15 \sqrt{15.78}}{(10-15.78)+.75} = \frac{.60}{-5.03} = -.119 \quad \phi = 173^\circ 12'$$

$$(\tan \phi)_{A=2} = \frac{.15 \sqrt{10.69}}{(10-10.69)+3} = \frac{.490}{2.31} = .213 \quad \phi = 12^\circ$$

$$= \frac{.15 \sqrt{15.29}}{(10-15.29)+3} = \frac{.590}{-2.29} = -.257 \quad \phi = 185^\circ 35'$$

$$(\tan \phi)_{A=0} = \frac{\infty}{-\infty^2} = -\frac{1}{\infty} = -0 \quad \phi = 180^\circ$$

$$(\tan \phi)_{A=1.5} = \frac{0}{(10-0)+.375} = 0 \quad \phi = 0^\circ$$



ω/ω_m	ϕ_1	ϕ_2
0	0	
.755	4° 4'	
1.036	12°	
1.25		173° 12'
1.24		185° 35'
∞		180°

14-38 Moment eq. about an accelerating point A is

$$\vec{M}_A = I_A \vec{\omega} + \vec{r}_{AC} \times m \vec{a}_A \quad \left(\begin{array}{l} \text{see Dynamics by} \\ \text{Pestel \& Thomson p 213} \\ \text{McGraw Hill.} \end{array} \right)$$

where C is the center of mass and \vec{r}_{AC} is a vector from A to C. For this problem $y_A = y_0 \cos 2\omega t$

$$I_A = ml^2 \quad |\vec{r}_{AC}| = l \quad |a_A| = -4y_0 \omega^2 \cos 2\omega t$$

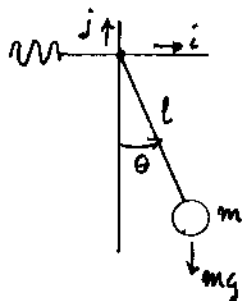
$$\vec{r}_{AC} \times m \vec{a}_A = l(\sin \theta \vec{i} - \cos \theta \vec{j}) \times m(-4y_0 \omega^2 \cos 2\omega t) \vec{j}$$

$$= -4y_0 \omega^2 l m \cos 2\omega t \cdot \vec{k}$$

$$\vec{\omega} = \ddot{\theta} \vec{k} \quad \vec{M}_A = -mgl \sin \theta \vec{k}$$

$$\therefore -mgl \sin \theta = ml^2 \ddot{\theta} - 4y_0 \omega^2 l m \cos 2\omega t$$

$$\ddot{\theta} + \left(\frac{g}{l} - \frac{4y_0 \omega^2 \cos 2\omega t}{l} \right) \sin \theta = 0$$



14-39

For the inverted pendulum $\theta = \pi - \phi$

$$\sin \theta = \sin \phi$$

$$\therefore -\ddot{\phi} + \left(\frac{g}{l} - \frac{4\omega_0^2 y_0}{l} \cos 2\omega t \right) \sin \phi = 0$$

For small ϕ

$$\ddot{\phi} + \left(-\frac{g}{l} + \frac{4\omega_0^2 y_0}{l} \cos 2\omega t \right) \phi = 0$$

Compare with Eq. (9) Sec. 14.5 which is

$$\frac{d^2 y}{dz^2} + (a - 2b \cos 2z) y = 0$$

which results in

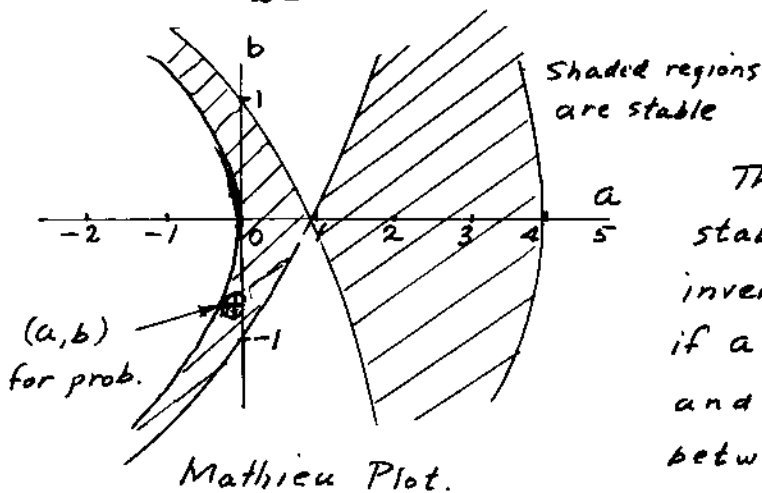
$$z = \omega t, \quad y = \phi, \quad dz^2 = \omega^2 dt$$

Rewrite eq.

$$\omega^2 \frac{d^2 \phi}{dz^2} + \left(-\frac{g}{l} + \frac{4\omega_0^2 y_0}{l} \cos 2z \right) \phi = 0$$

$$\frac{d^2 \phi}{dz^2} + \left(-\frac{g}{\omega^2 l} + \frac{4y_0}{l} \cos 2z \right) \phi = 0$$

$$\therefore a = -\frac{g}{\omega^2 l} \quad \text{and} \quad b = -\frac{2y_0}{l}$$



The plot indicates that stable oscillations of the inverted pendulum is possible if a is a small number and b is also negative between 0 & -1, as shown by the point \odot .

14-40 New length after displ is

$$l = l_0 \left(1 + \frac{x^2}{l_0^2}\right)^{1/2} \approx l_0 \left(1 + \frac{1}{2} \frac{x^2}{l_0^2}\right)$$



Let T_0 = initial tension. Since increase in length is $\frac{x^2}{l_0}$, the increase in tension is $K \frac{x^2}{l_0}$. Total tension = $(T_0 + K \frac{x^2}{l_0})$

Eq. of motion

$$m \ddot{x} = - \frac{x}{l} 2 \left(T_0 + K \frac{x^2}{l_0} \right) \quad \frac{x}{l} \approx \frac{x}{l_0}$$

$$m \ddot{x} + \left(\frac{2T_0}{l_0} \right) x + \left(\frac{2K}{l_0^2} \right) x^3 = 0$$

Assume sol. $x = x_1 + \mu x_2 + \dots$ where μ = arbitrary parameter and $x_2 \ll x_1$. Then $x^3 \approx x_1^3 + 3\mu x_1^2 x_2$. Subst. in D.E.

$$m \ddot{x}_1 + \frac{2T_0}{l_0} x_1 + \alpha x_1^3 = 0 \quad \alpha = \frac{2K}{l_0^2}$$

$$\mu \left[m \ddot{x}_2 + \frac{2T_0}{l_0} x_2 + 3\alpha x_1^2 x_2 \right] = 0$$

If α is small, then $x_1 = A \cos \omega_n t$, $\omega_n = \sqrt{\frac{2T_0}{l_0}}$ and

2nd eq. becomes

$$m \ddot{x}_2 + \frac{2T_0}{l_0} x_2 + (3\alpha A^2 \cos^2 \omega_n t) x_2 = 0$$

$$m \ddot{x}_2 + \left[\left(\frac{2T_0}{l_0} + \frac{3}{2} \alpha A^2 \right) + \frac{3}{2} \alpha A^2 \cos 2\omega_n t \right] x_2 = 0$$

which is Mathieu eq.

14-41

Runge - Kutta Program

$$\ddot{\theta} + \sin \theta = 0$$

$$\theta_0 = \frac{\pi}{3} = 60^\circ$$

$$\dot{\theta}_0 = 0$$

PROBLEM 14.41 THOMSON

```
DIMENSION T(100), T1(100), T2(100), T3(100), T4(100), X(100), X1(100),
1X2(100), X3(100), X4(100), Y(100), Y1(100), Y2(100), Y3(100), Y4(100),
1F(100), F1(100), F2(100), F3(100), F4(100)
N=70
DH=0.1
X(1)=3.1415/3.
Y(1)=0.3
T(1)=0.0
PRINT5
5 FORMAT(20X,'J',5X,'TIME',9X,'DISPL',5X,'ACCELERATION',11X,'F(J)')
DO 10 J=1,N
F(J)=FXY(T(J),X(J),Y(J))
PRINT8,J,T(J),X(J),Y(J),F(J)
8 FORMAT(18X,I3,2X,F7.3,2X,E12.3,5X,E12.3,3X,E12.3)
T1(J)=T(J)
X1(J)=X(J)
Y1(J)=Y(J)
F1(J)=FXY(T1(J),X1(J),Y1(J))
T2(J)=T(J)+DH/2.
X2(J)=X(J)+Y1(J)*DH/2.
Y2(J)=Y(J)+F1(J)*DH/2.
F2(J)=FXY(T2(J),X2(J),Y2(J))
T3(J)=T(J)+DH/2.
X3(J)=X(J)+Y2(J)*DH/2.
Y3(J)=Y(J)+F2(J)*DH/2.
F3(J)=FXY(T3(J),X3(J),Y3(J))
T4(J)=T(J)+DH
X4(J)=X(J)+Y3(J)*DH
Y4(J)=Y(J)+F3(J)*DH
F4(J)=FXY(T4(J),X4(J),Y4(J))
X(J+1)=Y(J)+DH/6.*(Y1(J)+2.*Y2(J)+2.*Y3(J)+Y4(J))
Y(J+1)=Y(J)+DH/6.*(F1(J)+2.*F2(J)+2.*F3(J)+F4(J))
T(J+1)=T(J)+DH
10 CONTINUE
STOP
END

FUNCTION FXY(T,X,Y)
FXY=-SIN(X)
RETURN
END
```

J	TIME	DISPL	ACCELERATION	F (J)
1	0.000	0.105E 01	0.000E 00	-0.866E 00
2	0.100	0.104E 01	-0.865E-01	-0.864E 00
3	0.200	0.103E 01	-0.173E 00	-0.857E 00
4	0.300	0.101E 01	-0.258E 00	-0.846E 00
5	0.400	0.978E 00	-0.342E 00	-0.830E 00
6	0.500	0.940E 00	-0.424E 00	-0.808E 00
7	0.600	0.894E 00	-0.503E 00	-0.779E 00
8	0.700	0.840E 00	-0.579E 00	-0.744E 00
9	0.800	0.778E 00	-0.652E 00	-0.702E 00
10	0.900	0.709E 00	-0.719E 00	-0.651E 00
11	1.000	0.634E 00	-0.782E 00	-0.593E 00
12	1.100	0.553E 00	-0.838E 00	-0.526E 00
13	1.200	0.467E 00	-0.886E 00	-0.450E 00
14	1.300	0.376E 00	-0.927E 00	-0.367E 00
15	1.400	0.282E 00	-0.960E 00	-0.278E 00
16	1.500	0.185E 00	-0.983E 00	-0.184E 00
17	1.600	0.856E-01	-0.996E 00	-0.855E-01
18	1.700	-0.143E-01	-0.100E 01	0.143E-01
19	1.800	-0.114E 00	-0.993E 00	0.114E 00
20	1.900	-0.213E 00	-0.977E 00	0.211E 00
21	2.000	-0.309E 00	-0.951E 00	0.304E 00
22	2.100	-0.403E 00	-0.917E 00	0.392E 00
23	2.200	-0.492E 00	-0.873E 00	0.473E 00
24	2.300	-0.577E 00	-0.822E 00	0.546E 00
25	2.400	-0.656E 00	-0.764E 00	0.610E 00
26	2.500	-0.730E 00	-0.701E 00	0.667E 00
27	2.600	-0.796E 00	-0.631E 00	0.715E 00
28	2.700	-0.856E 00	-0.558E 00	0.755E 00
29	2.800	-0.909E 00	-0.481E 00	0.788E 00
30	2.900	-0.952E 00	-0.400E 00	0.814E 00
31	3.000	-0.988E 00	-0.318E 00	0.835E 00
32	3.100	-0.102E 01	-0.234E 00	0.850E 00
33	3.200	-0.103E 01	-0.148E 00	0.860E 00
34	3.300	-0.104E 01	-0.619E-01	0.865E 00
35	3.400	-0.105E 01	0.247E-01	0.866E 00
36	3.500	-0.104E 01	0.111E 00	0.862E 00
37	3.600	-0.102E 01	0.197E 00	0.855E 00
38	3.700	-0.100E 01	0.282E 00	0.842E 00
39	3.800	-0.968E 00	0.365E 00	0.824E 00
40	3.900	-0.929E 00	0.446E 00	0.800E 00

41	4.000	-0.879E 00	0.525E 00	0.770E 00
42	4.100	-0.923E 00	0.600E 00	0.733E 00
43	4.200	-0.759E 00	0.671E 00	0.688E 00
44	4.300	-0.689E 00	0.738E 00	0.636E 00
45	4.400	-0.612E 00	0.798E 00	0.574E 00
46	4.500	-0.529E 00	0.852E 00	0.505E 00
47	4.600	-0.442E 00	0.899E 00	0.427E 00
48	4.700	-0.350E 00	0.937E 00	0.343E 00
49	4.800	-0.254E 00	0.967E 00	0.252E 00
50	4.900	-0.157E 00	0.988E 00	0.156E 00
51	5.000	-0.572E-01	0.998E 00	0.572E-01
52	5.100	0.428E-01	0.999E 00	-0.427E-01
53	5.200	0.142E 00	0.990E 00	-0.142E 00
54	5.300	0.240E 00	0.971E 00	-0.238E 00
55	5.400	0.336E 00	0.942E 00	-0.330E 00
56	5.500	0.429E 00	0.905E 00	-0.416E 00
57	5.600	0.517E 00	0.859E 00	-0.494E 00
58	5.700	0.600E 00	0.806E 00	-0.565E 00
59	5.800	0.678E 00	0.747E 00	-0.627E 00
60	5.900	0.749E 00	0.681E 00	-0.681E 00
61	6.000	0.814E 00	0.611E 00	-0.727E 00
62	6.100	0.871E 00	0.536E 00	-0.765E 00
63	6.200	0.921E 00	0.458E 00	-0.796E 00
64	6.300	0.963E 00	0.377E 00	-0.821E 00
65	6.400	0.996E 00	0.294E 00	-0.840E 00
66	6.500	0.102E 01	0.209E 00	-0.853E 00
67	6.600	0.104E 01	0.124E 00	-0.862E 00
68	6.700	0.105E 01	0.372E-01	-0.866E 00
69	6.800	0.105E 01	-0.494E-01	-0.865E 00
70	6.900	0.104E 01	-0.136E 00	-0.861E 00

14-42

Damped Pendulum (large angles)

$$\ddot{\theta} + \sin \theta + 0.30 \dot{\theta} = 0, \quad \frac{g}{L} = 1.0$$

$$\theta(0) = \frac{\pi}{3} = 60^\circ, \quad \dot{\theta}(0) = 0$$

Runge-Kutta Program

PROBLEM 14.42 THOMSON

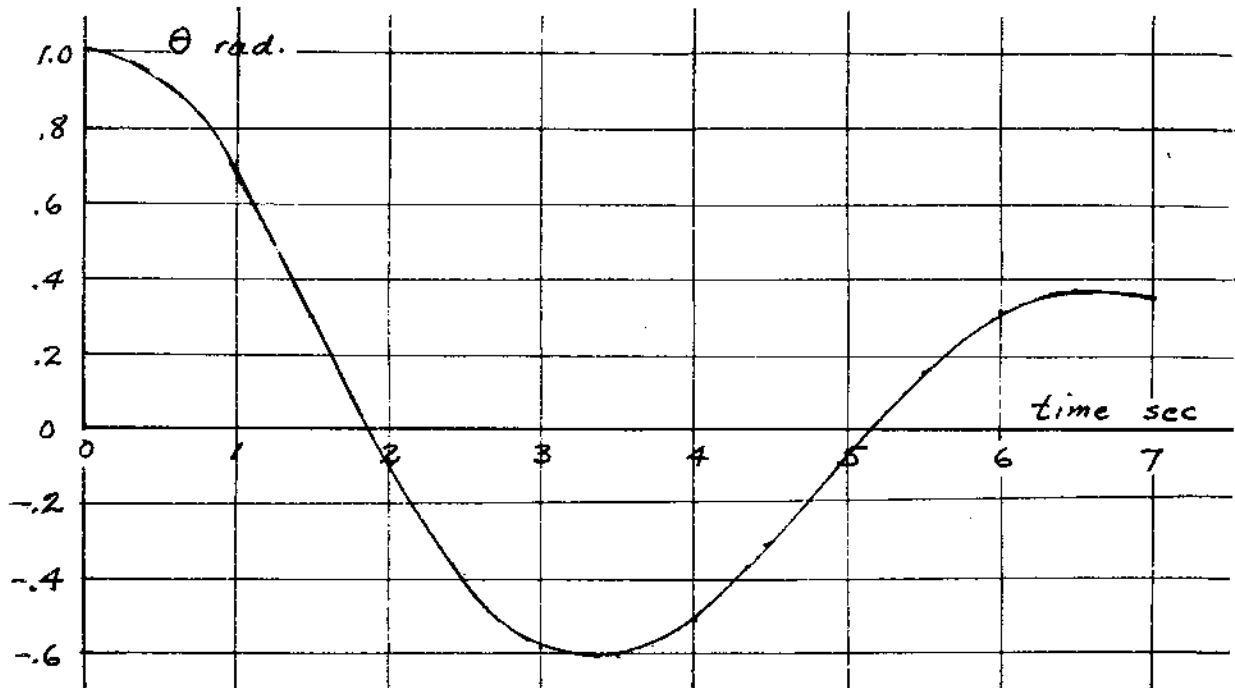
```
DIMENSION T(100), T1(100), T2(100), T3(100), T4(100), X(100), X1(100),
1 X2(100), X3(100), X4(100), Y(100), Y1(100), Y2(100), Y3(100), Y4(100),
1 F(100), F1(100), F2(100), F3(100), F4(100)
N=70
DH=0.1
X(1)=3.1415/3.
Y(1)=0.0
T(1)=0.0
PRINT5
5 FORMAT(20X,'J',5X,'TIME',9X,'DISPL',5X,'ACCELERATION',11X,'F(J)')
DO 10 J=1,N
  F(J)=FXY(T(J),X(J),Y(J))
  PRINT8,1,T(J),X(J),Y(J),F(J)
8 FORMAT(18X,I3,2X,F7.3,2X,E12.3,5X,E12.3,3X,E12.3)
  T1(J)=T(J)
  X1(J)=X(J)
  Y1(J)=Y(J)
  F1(J)=FXY(T1(J),X1(J),Y1(J))
  T2(J)=T(J)+DH/2.
  X2(J)=X(J)+Y1(J)*DH/2.
  Y2(J)=Y(J)+F1(J)*DH/2.
  F2(J)=FXY(T2(J),X2(J),Y2(J))
  T3(J)=T(J)+DH/2.
  X3(J)=X(J)+Y2(J)*DH/2.
  Y3(J)=Y(J)+F2(J)*DH/2.
  F3(J)=FXY(T3(J),X3(J),Y3(J))
  T4(J)=T(J)+DH
  X4(J)=X(J)+Y3(J)*DH
  Y4(J)=Y(J)+F3(J)*DH
  F4(J)=FXY(T4(J),X4(J),Y4(J))
  X(J+1)=X(J)+DH/6.*(Y1(J)+2.*Y2(J)+2.*Y3(J)+Y4(J))
  Y(J+1)=Y(J)+DH/6.*(F1(J)+2.*F2(J)+2.*F3(J)+F4(J))
  T(J+1)=T(J)+DH
10 CONTINUE
STOP
END

FUNCTION FXY(T,X,Y)
FXY=-SIN(X)-0.3*Y
RETURN
END
```

J	TIME	DISPL	ACCELERATION	F (J)
1	0.000	0.105E 01	0.000E 00	-0.866E 00
2	0.100	0.104E 01	-0.852E-01	-0.838E 00
3	0.200	0.103E 01	-0.168E 00	-0.807E 00
4	0.300	0.101E 01	-0.247E 00	-0.773E 00
5	0.400	0.981E 00	-0.322E 00	-0.734E 00
6	0.500	0.945E 00	-0.393E 00	-0.693E 00
7	0.600	0.903E 00	-0.460E 00	-0.647E 00
8	0.700	0.853E 00	-0.523E 00	-0.597E 00
9	0.800	0.798E 00	-0.580E 00	-0.542E 00
10	0.900	0.738E 00	-0.631E 00	-0.483E 00
11	1.000	0.672E 00	-0.676E 00	-0.420E 00
12	1.100	0.603E 00	-0.715E 00	-0.352E 00
13	1.200	0.530E 00	-0.746E 00	-0.281E 00
14	1.300	0.454E 00	-0.771E 00	-0.207E 00
15	1.400	0.376E 00	-0.788E 00	-0.131E 00
16	1.500	0.296E 00	-0.797E 00	-0.529E-01
17	1.600	0.216E 00	-0.798E 00	0.247E-01
18	1.700	0.137E 00	-0.792E 00	0.101E 00
19	1.800	0.593E-01	-0.778E 00	0.175E 00
20	1.900	-0.185E-01	-0.757E 00	0.246E 00
21	2.000	-0.929E-01	-0.729E 00	0.311E 00
22	2.100	-0.164E 00	-0.695E 00	0.372E 00
23	2.200	-0.232E 00	-0.655E 00	0.426E 00
24	2.300	-0.295E 00	-0.610E 00	0.474E 00
25	2.400	-0.354E 00	-0.561E 00	0.514E 00
26	2.500	-0.407E 00	-0.507E 00	0.548E 00
27	2.600	-0.455E 00	-0.451E 00	0.575E 00
28	2.700	-0.497E 00	-0.393E 00	0.595E 00
29	2.800	-0.533E 00	-0.332E 00	0.608E 00
30	2.900	-0.564E 00	-0.271E 00	0.616E 00
31	3.000	-0.588E 00	-0.210E 00	0.617E 00
32	3.100	-0.605E 00	-0.148E 00	0.613E 00
33	3.200	-0.617E 00	-0.870E-01	0.605E 00
34	3.300	-0.623E 00	-0.271E-01	0.591E 00
35	3.400	-0.623E 00	0.312E-01	0.574E 00
36	3.500	-0.617E 00	0.875E-01	0.552E 00
37	3.600	-0.605E 00	0.141E 00	0.527E 00
38	3.700	-0.588E 00	0.193E 00	0.497E 00
39	3.800	-0.567E 00	0.241E 00	0.465E 00
40	3.900	-0.540E 00	0.286E 00	0.429E 00
41	4.000	-0.510E 00	0.326E 00	0.390E 00
42	4.100	-0.475E 00	0.363E 00	0.349E 00

14-42 Cont:

43	4.200	-0.437E 00	0.396E 00	0.305E 00
44	4.300	-0.396E 00	0.424E 00	0.259E 00
45	4.400	-0.353E 00	0.448E 00	0.211E 00
46	4.500	-0.307E 00	0.466E 00	0.162E 00
47	4.600	-0.259E 00	0.480E 00	0.112E 00
48	4.700	-0.211E 00	0.489E 00	0.627E-01
49	4.800	-0.162E 00	0.493E 00	0.133E-01
50	4.900	-0.113E 00	0.492E 00	-0.352E-01
51	5.000	-0.636E-01	0.486E 00	-0.821E-01
52	5.100	-0.156E-01	0.475E 00	-0.127E 00
53	5.200	0.313E-01	0.460E 00	-0.169E 00
54	5.300	0.764E-01	0.441E 00	-0.209E 00
55	5.400	0.119E 00	0.419E 00	-0.245E 00
56	5.500	0.160E 00	0.393E 00	-0.277E 00
57	5.600	0.198E 00	0.363E 00	-0.306E 00
58	5.700	0.233E 00	0.332E 00	-0.330E 00
59	5.800	0.264E 00	0.298E 00	-0.350E 00
60	5.900	0.292E 00	0.262E 00	-0.366E 00
61	6.000	0.316E 00	0.224E 00	-0.379E 00
62	6.100	0.337E 00	0.186E 00	-0.386E 00
63	6.200	0.354E 00	0.147E 00	-0.390E 00
64	6.300	0.366E 00	0.108E 00	-0.391E 00
65	6.400	0.375E 00	0.693E-01	-0.387E 00
66	6.500	0.380E 00	0.309E-01	-0.380E 00
67	6.600	0.381E 00	-0.671E-02	-0.370E 00
68	6.700	0.379E 00	-0.431E-01	-0.357E 00
69	6.800	0.373E 00	-0.780E-01	-0.341E 00
70	6.900	0.363E 00	-0.111E 00	-0.322E 00

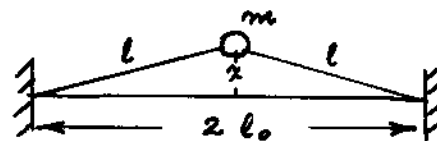


14-43

Runge-Kutta Program

$$\ddot{\xi} + \xi + 0.2 \xi^3 = 0$$

$$\xi = \frac{x}{l_0}, \quad \xi(0) = 0.20, \quad \dot{\xi}(0) = 0$$



Runge-Kutta Program

PROBLEM 14.43 THOMSON

```

DIMENSION T(100), T1(100), T2(100), T3(100), T4(100), X(100), X1(100),
Y(100), Y1(100), Y2(100), Y3(100), Y4(100),
F(100), F1(100), F2(100), F3(100), F4(100)
N=70
DH=0.1
X(1)=0.2
Y(1)=0.0
T(1)=0.0
PRINT5
5 FORMAT(20X,'J',5X,'TIME',9X,'DISPL',5X,'ACCELERATION',11X,'F(J)')
DO 10 J=1,N
  F(J)=FXY(T(J),X(J),Y(J))
  PRINT8,J,T(J),X(J),Y(J),F(J)
8 FORMAT(18X,I3,2X,F7.3,2X,E12.3,5X,E12.3,3X,E12.3)
  T1(J)=T(J)
  X1(J)=X(J)
  Y1(J)=Y(J)
  F1(J)=FXY(T1(J),X1(J),Y1(J))
  T2(J)=T(J)+DH/2.
  X2(J)=X(J)+Y1(J)*DH/2.
  Y2(J)=Y(J)+F1(J)*DH/2.
  F2(J)=FXY(T2(J),X2(J),Y2(J))
  T3(J)=T(J)+DH/2.
  X3(J)=X(J)+Y2(J)*DH/2.
  Y3(J)=Y(J)+F2(J)*DH/2.
  F3(J)=FXY(T3(J),X3(J),Y3(J))
  T4(J)=T(J)+DH
  X4(J)=X(J)+Y3(J)*DH
  Y4(J)=Y(J)+F3(J)*DH
  F4(J)=FXY(T4(J),X4(J),Y4(J))
  X(J+1)=X(J)+DH/6.*(Y1(J)+2.*Y2(J)+2.*Y3(J)+Y4(J))
  Y(J+1)=Y(J)+DH/6.*(F1(J)+2.*F2(J)+2.*F3(J)+F4(J))
  T(J+1)=T(J)+DH
10 CONTINUE
STOP
END

FUNCTION FXY(T,X,Y)
  FXY=-Y-0.2*X**3
RETURN
END

```

J	TIME	DISPL	ACCELERATION	F (J)
1	0.000	0.200E 00	0.000E 00	-0.202E 00
2	0.100	0.199E 00	-0.201E-01	-0.201E 00
3	0.200	0.196E 00	-0.400E-01	-0.197E 00
4	0.300	0.191E 00	-0.596E-01	-0.192E 00
5	0.400	0.184E 00	-0.785E-01	-0.185E 00
6	0.500	0.175E 00	-0.966E-01	-0.176E 00
7	0.600	0.165E 00	-0.114E 00	-0.166E 00
8	0.700	0.153E 00	-0.130E 00	-0.153E 00
9	0.800	0.139E 00	-0.144E 00	-0.139E 00
10	0.900	0.124E 00	-0.157E 00	-0.124E 00
11	1.000	0.107E 00	-0.169E 00	-0.108E 00
12	1.100	0.901E-01	-0.179E 00	-0.902E-01
13	1.200	0.717E-01	-0.187E 00	-0.718E-01
14	1.300	0.527E-01	-0.193E 00	-0.527E-01
15	1.400	0.331E-01	-0.198E 00	-0.331E-01
16	1.500	0.132E-01	-0.200E 00	-0.132E-01
17	1.600	-0.679E-02	-0.200E 00	0.679E-02
18	1.700	-0.268E-01	-0.199E 00	0.268E-01
19	1.800	-0.464E-01	-0.195E 00	0.465E-01
20	1.900	-0.657E-01	-0.189E 00	0.657E-01
21	2.000	-0.842E-01	-0.182E 00	0.844E-01
22	2.100	-0.102E 00	-0.172E 00	0.102E 00
23	2.200	-0.119E 00	-0.161E 00	0.119E 00
24	2.300	-0.134E 00	-0.149E 00	0.135E 00
25	2.400	-0.148E 00	-0.135E 00	0.149E 00
26	2.500	-0.161E 00	-0.119E 00	0.162E 00
27	2.600	-0.172E 00	-0.102E 00	0.173E 00
28	2.700	-0.181E 00	-0.844E-01	0.183E 00
29	2.800	-0.189E 00	-0.657E-01	0.190E 00
30	2.900	-0.195E 00	-0.464E-01	0.196E 00
31	3.000	-0.198E 00	-0.266E-01	0.200E 00
32	3.100	-0.200E 00	-0.649E-02	0.201E 00
33	3.200	-0.200E 00	0.137E-01	0.201E 00
34	3.300	-0.197E 00	0.337E-01	0.199E 00
35	3.400	-0.193E 00	0.533E-01	0.194E 00
36	3.500	-0.187E 00	0.724E-01	0.188E 00
37	3.600	-0.178E 00	0.908E-01	0.179E 00
38	3.700	-0.168E 00	0.103E 00	0.169E 00
39	3.800	-0.157E 00	0.125E 00	0.158E 00
40	3.900	-0.144E 00	0.140E 00	0.144E 00
41	4.000	-0.129E 00	0.153E 00	0.129E 00
42	4.100	-0.113E 00	0.166E 00	0.113E 00

14-43 Cont:

	time	displ.	accel.	F(J)
43	4.200	-0.958E-01	0.176E 00	0.960E-01
44	4.300	-0.777E-01	0.185E 00	0.778E-01
45	4.400	-0.589E-01	0.192E 00	0.589E-01
46	4.500	-0.395E-01	0.196E 00	0.395E-01
47	4.600	-0.197E-01	0.199E 00	0.197E-01
48	4.700	0.335E-03	0.200E 00	-0.335E-03
49	4.800	0.203E-01	0.199E 00	-0.203E-01
50	4.900	0.401E-01	0.196E 00	-0.402E-01
51	5.000	0.595E-01	0.191E 00	-0.596E-01
52	5.100	0.783E-01	0.184E 00	-0.784E-01
53	5.200	0.964E-01	0.176E 00	-0.965E-01
54	5.300	0.113E 00	0.165E 00	-0.114E 00
55	5.400	0.129E 00	0.153E 00	-0.130E 00
56	5.500	0.144E 00	0.139E 00	-0.145E 00
57	5.600	0.157E 00	0.124E 00	-0.158E 00
58	5.700	0.169E 00	0.108E 00	-0.170E 00
59	5.800	0.179E 00	0.902E-01	-0.180E 00
60	5.900	0.187E 00	0.718E-01	-0.188E 00
61	6.000	0.193E 00	0.527E-01	-0.194E 00
62	6.100	0.197E 00	0.330E-01	-0.199E 00
63	6.200	0.200E 00	0.130E-01	-0.201E 00
64	6.300	0.200E 00	-0.717E-02	-0.201E 00
65	6.400	0.198E 00	-0.272E-01	-0.200E 00
66	6.500	0.194E 00	-0.470E-01	-0.196E 00
67	6.600	0.189E 00	-0.664E-01	-0.190E 00
68	6.700	0.181E 00	-0.850E-01	-0.182E 00
69	6.800	0.172E 00	-0.103E 00	-0.173E 00
70	6.900	0.161E 00	-0.119E 00	-0.161E 00

