

Finite-Difference Schemes (pt. 2)

- Physical Models for Sound Synthesis -

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Agenda



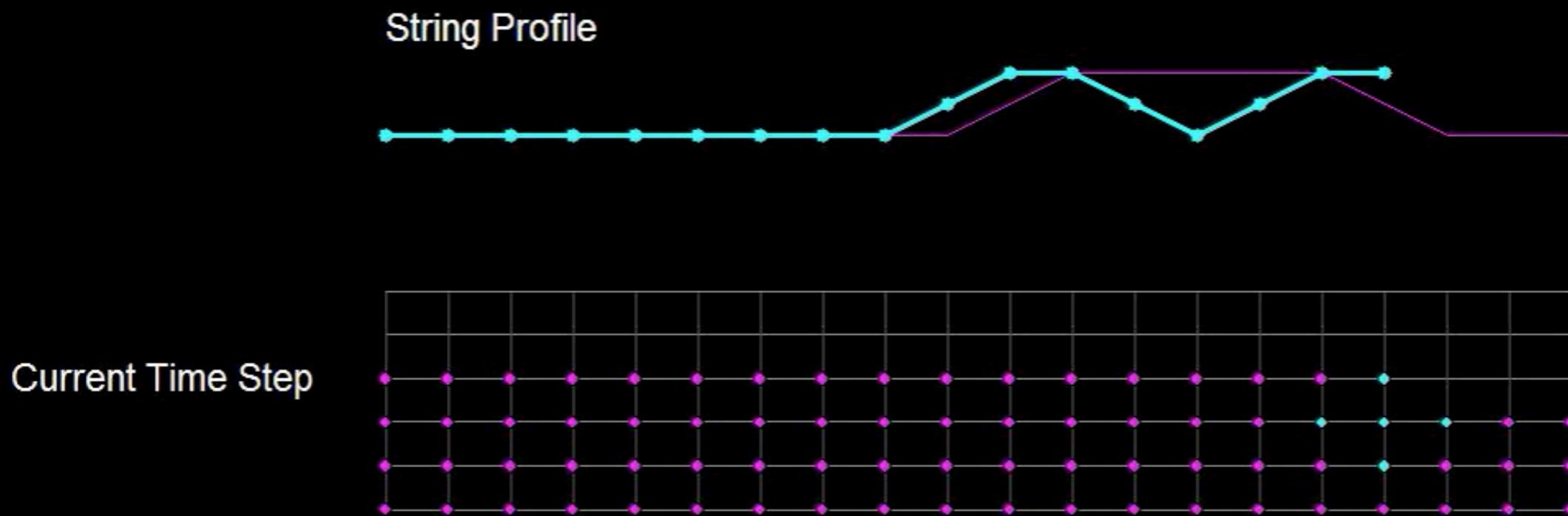
- Recap last lecture
- Damped 1D wave
- Ideal Bar
- (Damped) Stiff String
- Towards complete instruments
 - ↳ Connecting models

Recap



Recap

- FDSs subdivide continuous systems into points in space and time.

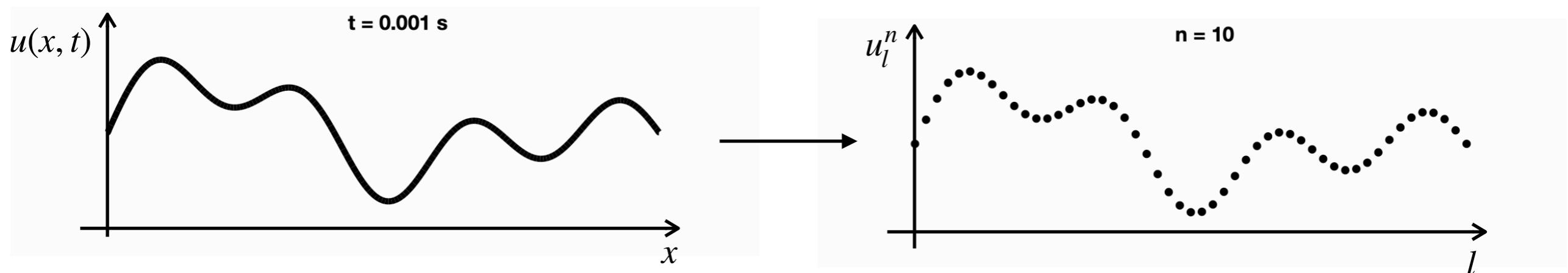


University of Edinburgh, NESS: Next Generation Sound Synthesis, 2016, url: <http://www.ness.music.ed.ac.uk/project#3>



Recap

- FDSs subdivide continuous systems into points in space and time.
- The state of a (1D) system is denoted by $u = u(x, t)$ which becomes grid function u_l^n when discretised.





Recap

- FDSs subdivide continuous systems into points in space and time.
- The state of a (1D) system is denoted by $u = u(x, t)$ which becomes grid function u_l^n when discretised.
- Steps to implementing a (continuous) PDE:
 1. Discretise PDE to FDS. $u_{tt} = \dots \rightarrow \delta_{tt} u_l^n = \dots$
 2. Expand operators. $\rightarrow \frac{1}{k^2} (u_l^{n+1} - 2u_l^n + u_l^{n-1}) = \dots$
 3. Solve for u_l^{n+1} $\rightarrow u_l^{n+1} = 2u_l^n - u_l^{n-1} + k^2(\dots)$
 4. Define domain and boundary conditions. $N = \dots \quad l = [0, \dots, N]$
 $u_0^n = u_N^n = 0$

**Damped 1D wave
equation**

Damped 1D wave equation



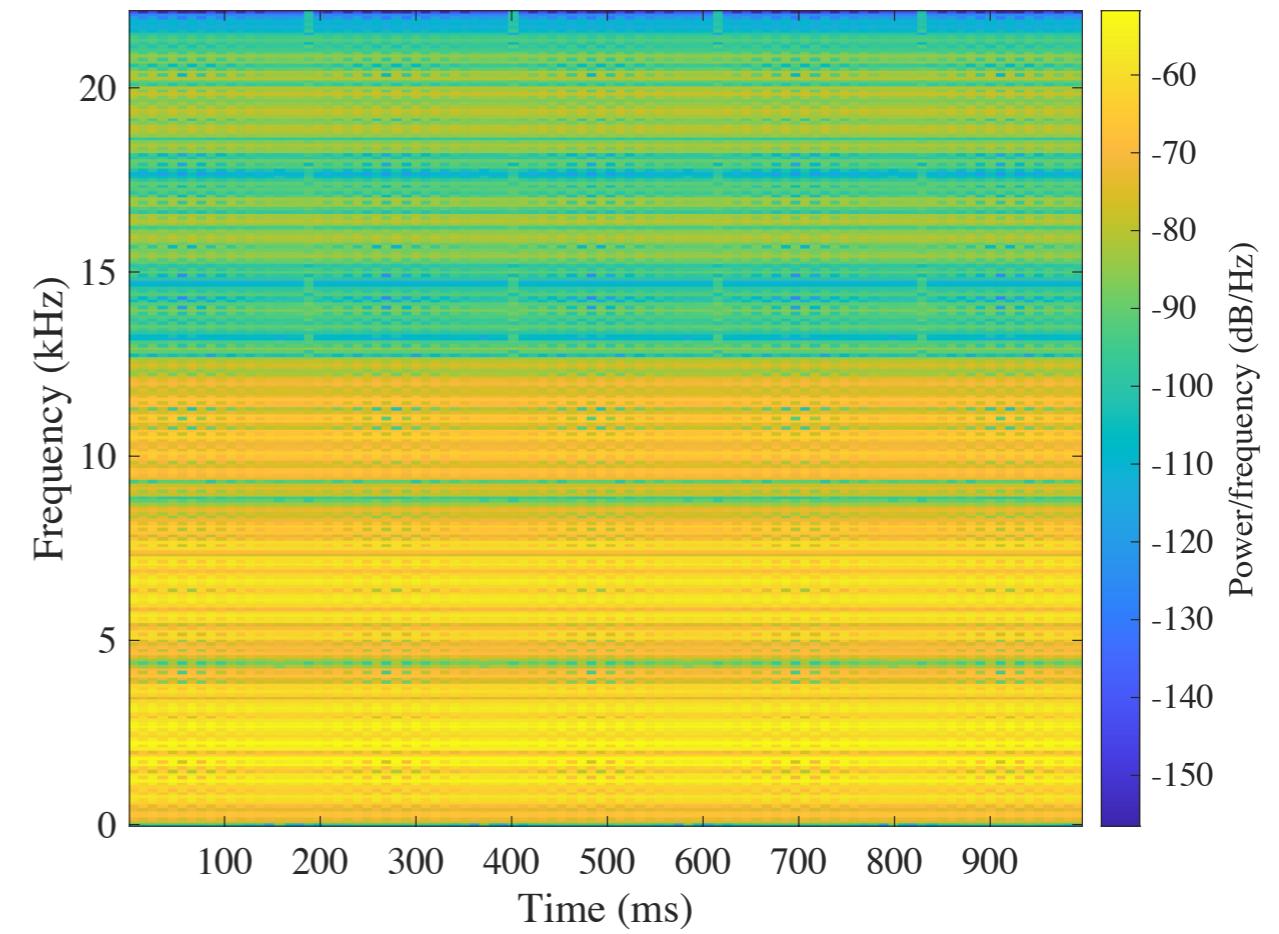
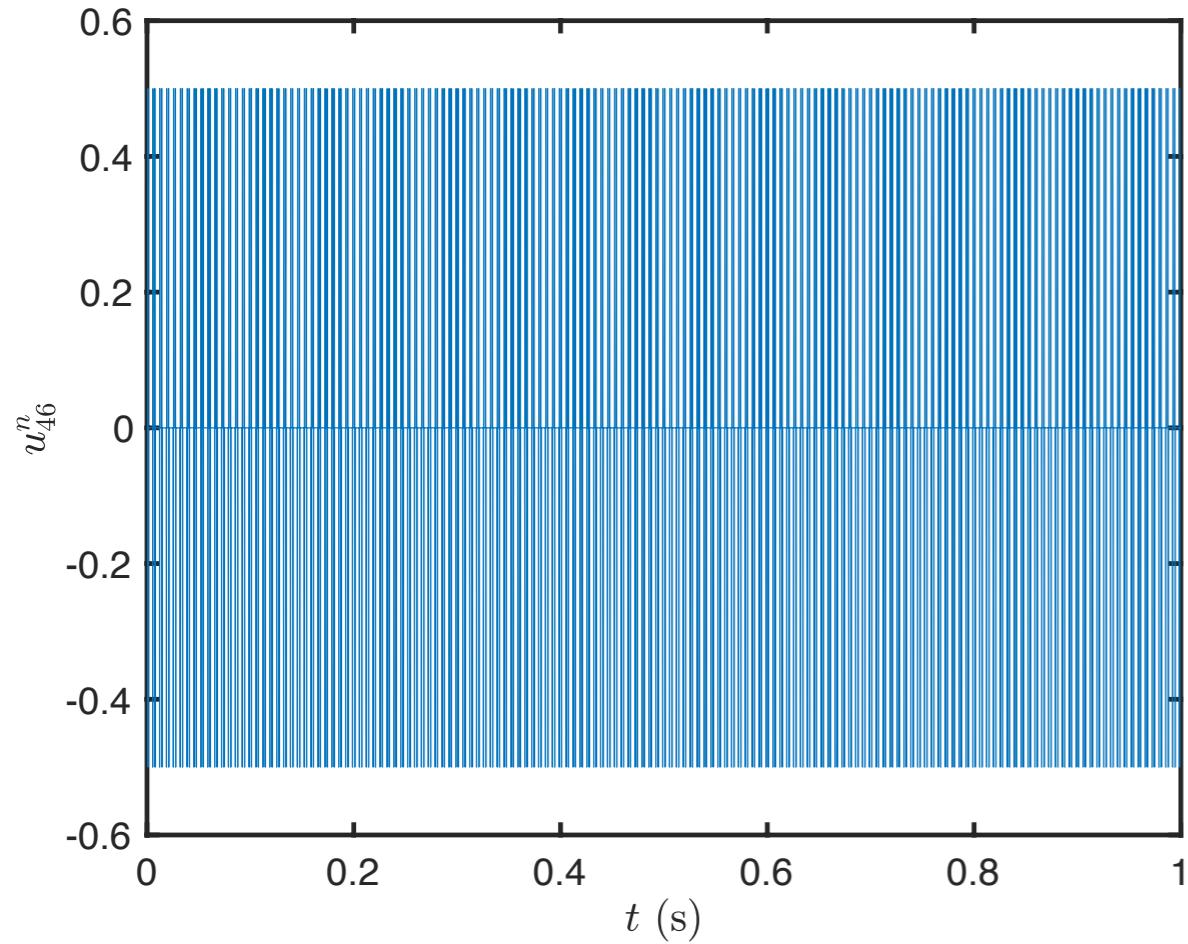
Who wants to show their implementation? :)

Damped 1D wave equation



Sound examples

$$c = 300 \quad L = 1 \quad \sigma_0 = 0 \quad \sigma_1 = 0$$

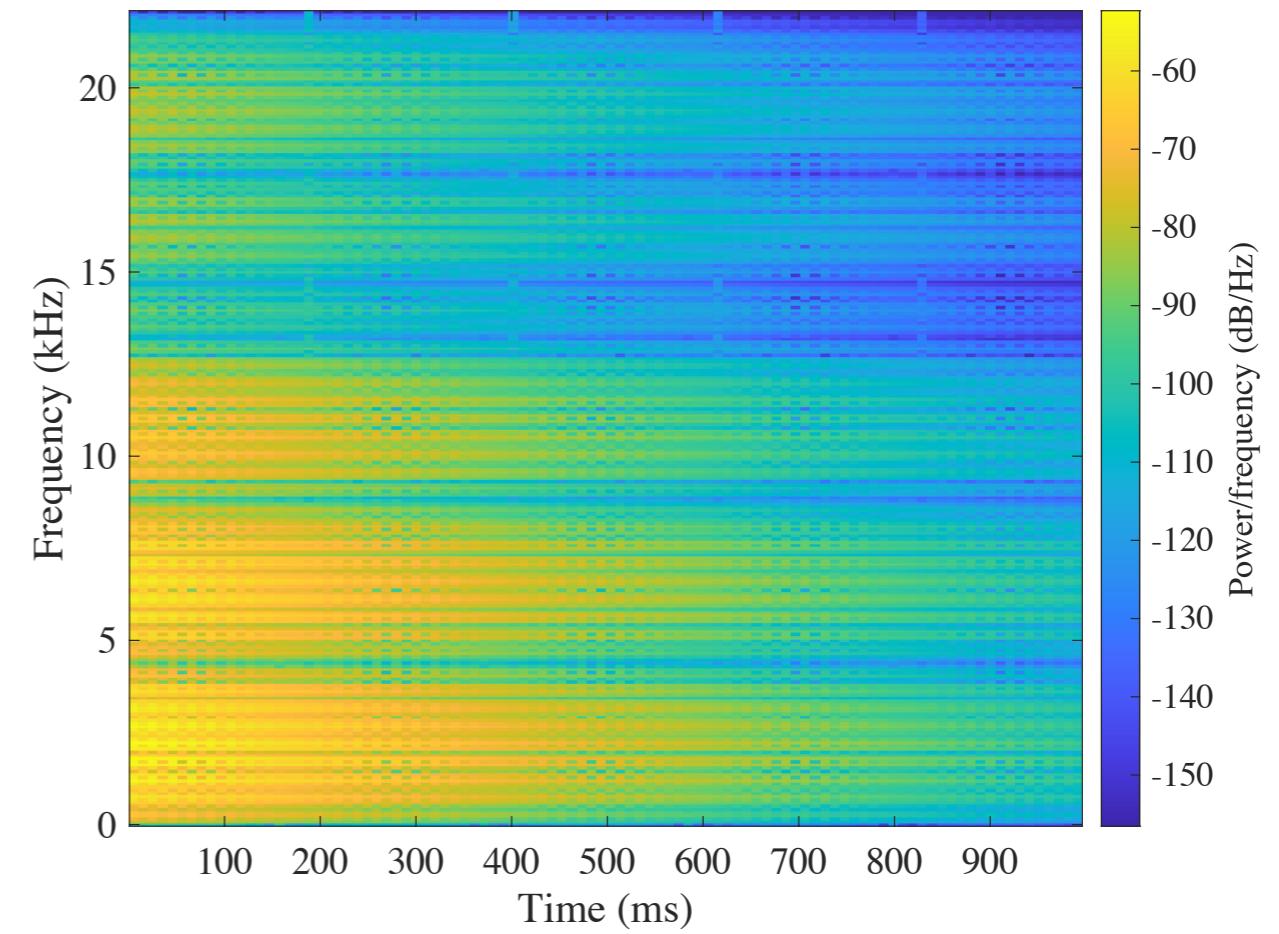
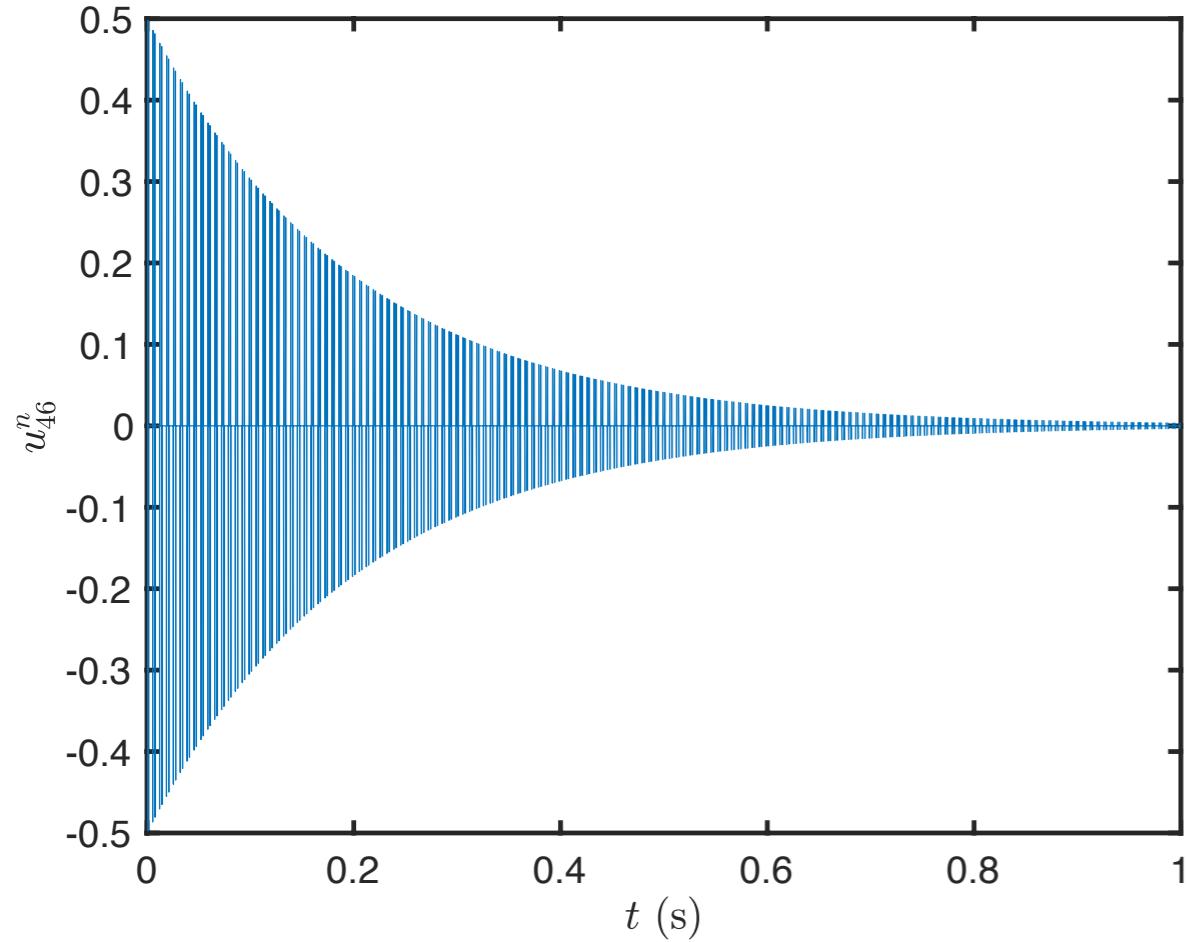


Damped 1D wave equation



Sound examples

$$c = 300 \quad L = 1 \quad \sigma_0 = 5 \quad \sigma_1 = 0$$

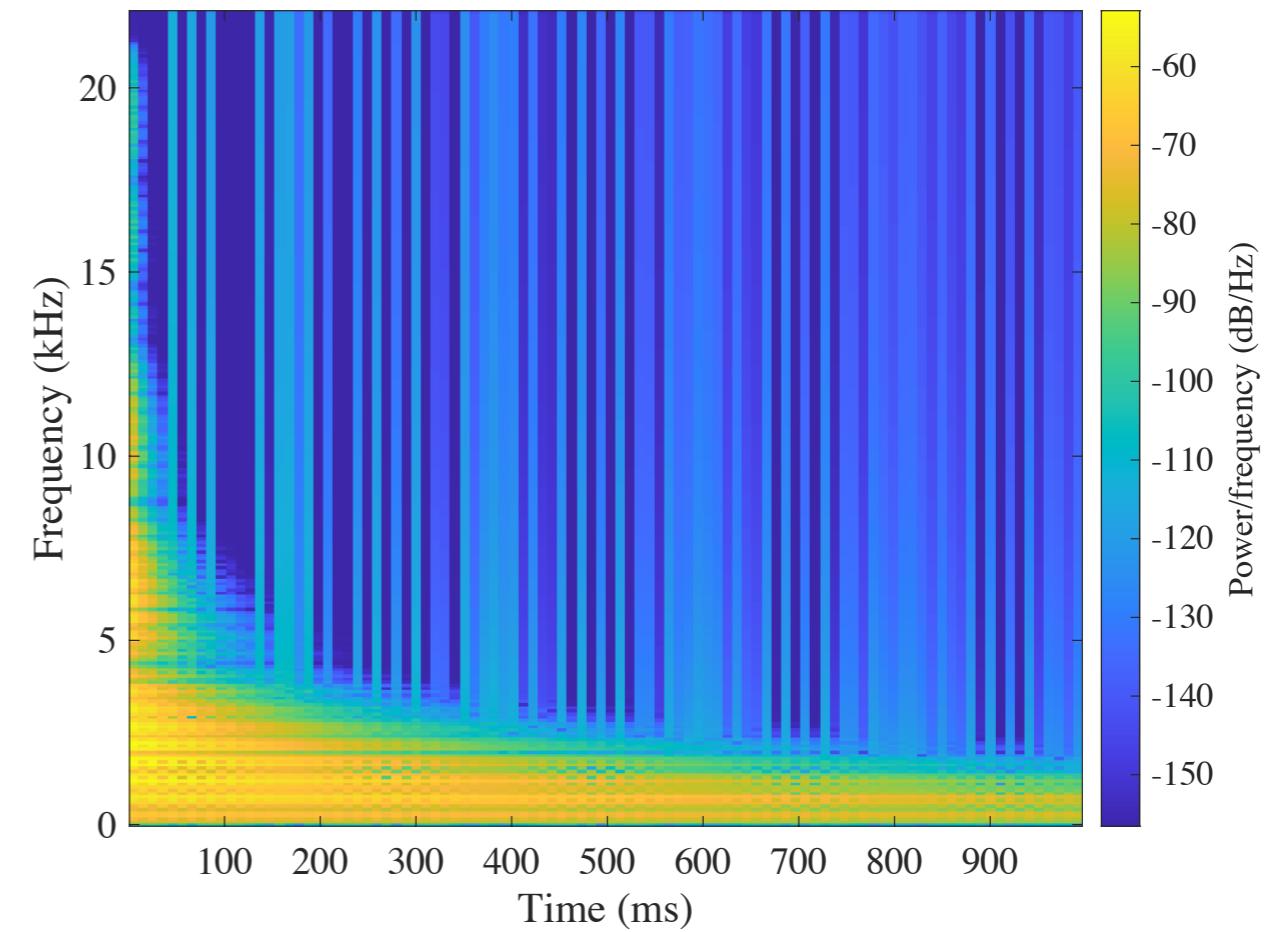
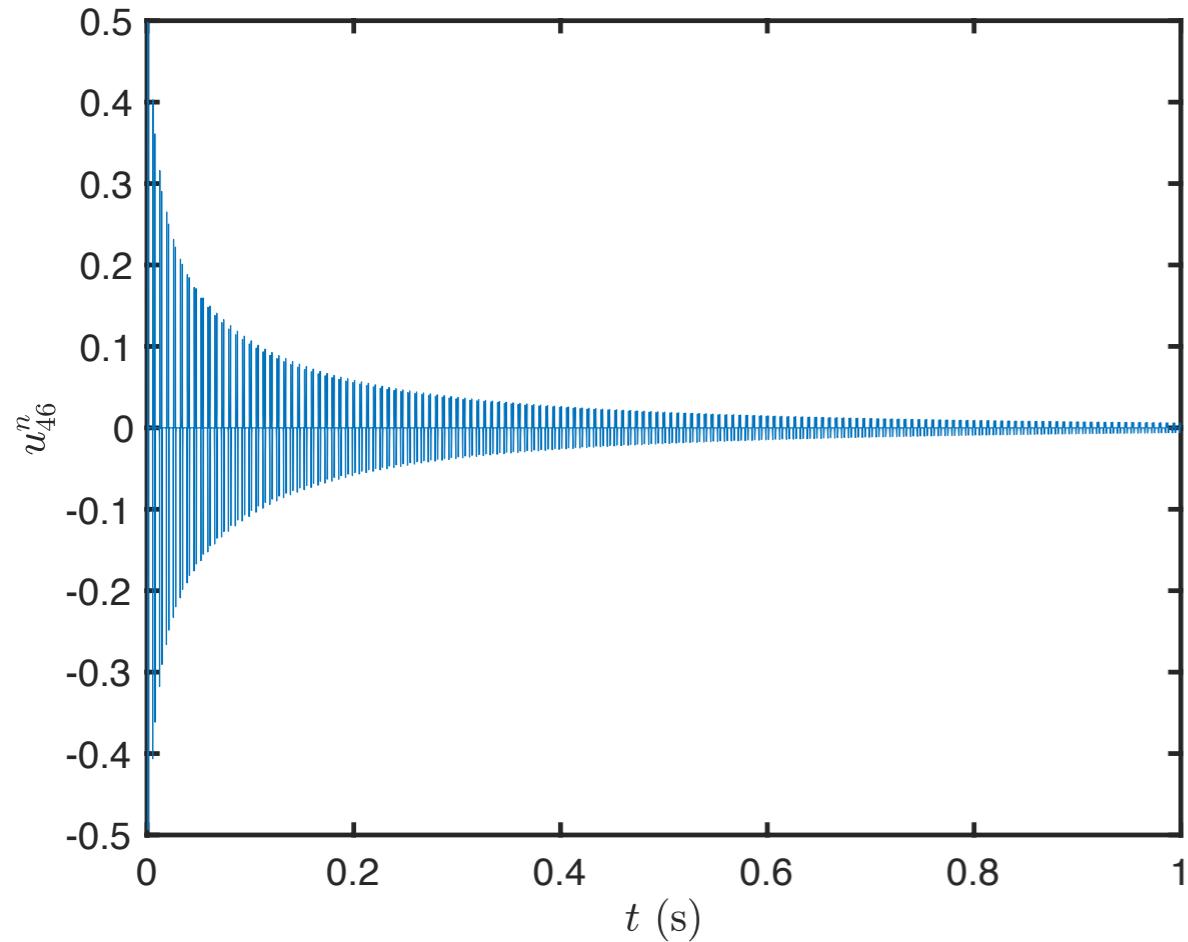


Damped 1D wave equation



Sound examples

$$c = 300 \quad L = 1 \quad \sigma_0 = 1 \quad \sigma_1 = 0.005$$





Damped 1D wave equation

What choices do we have for discretising

$$u_{tt} = c^2 u_{xx} - 2\sigma_0 u_t + 2\sigma_1 u_{txx}$$

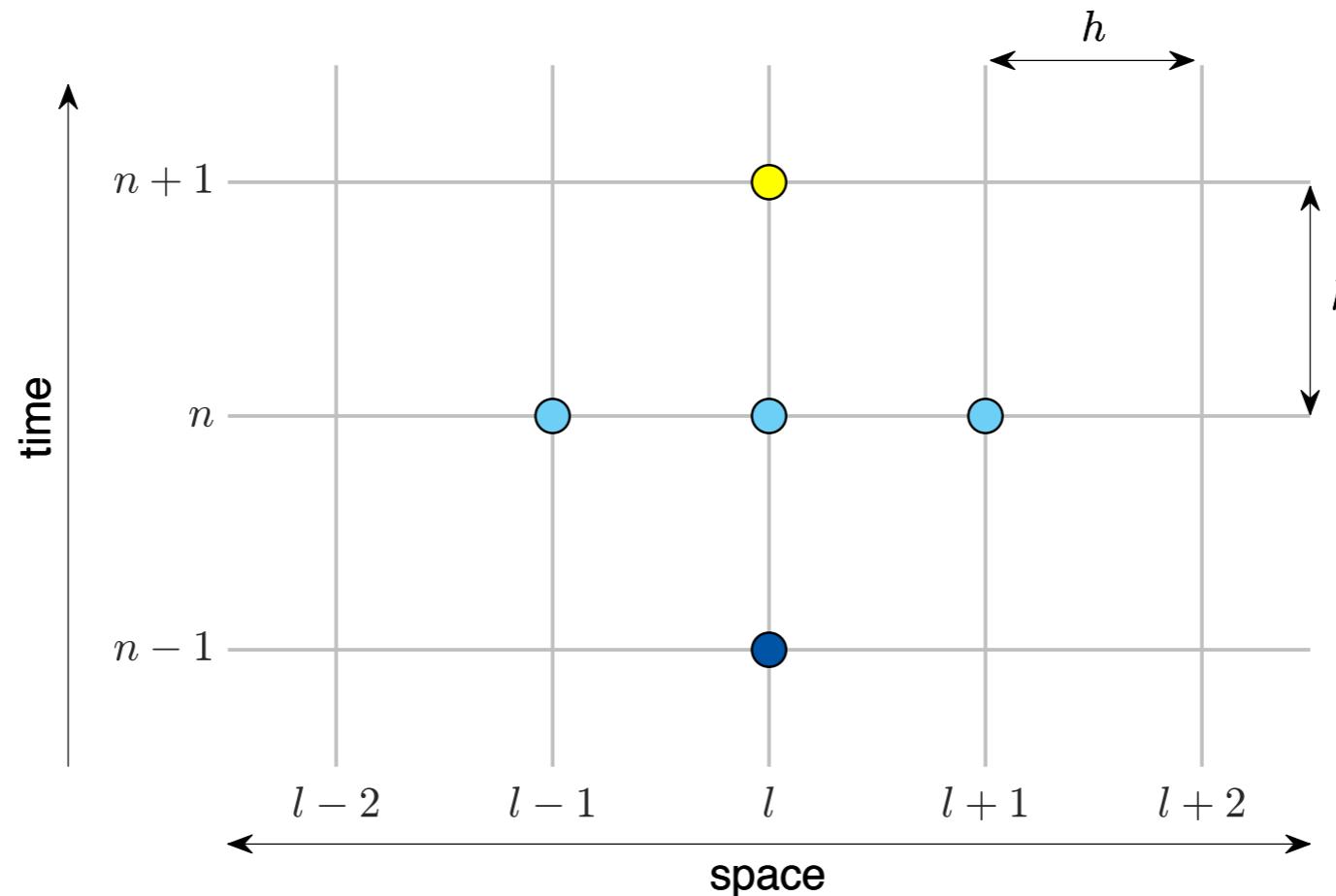
and why?



Damped 1D wave equation

It can be useful to talk about a FDS in terms of a **stencil**, or the 'region of operation'.

The stencil of $\delta_{tt}u_l^n = c^2\delta_{xx}u_l^n$ can be visualised as



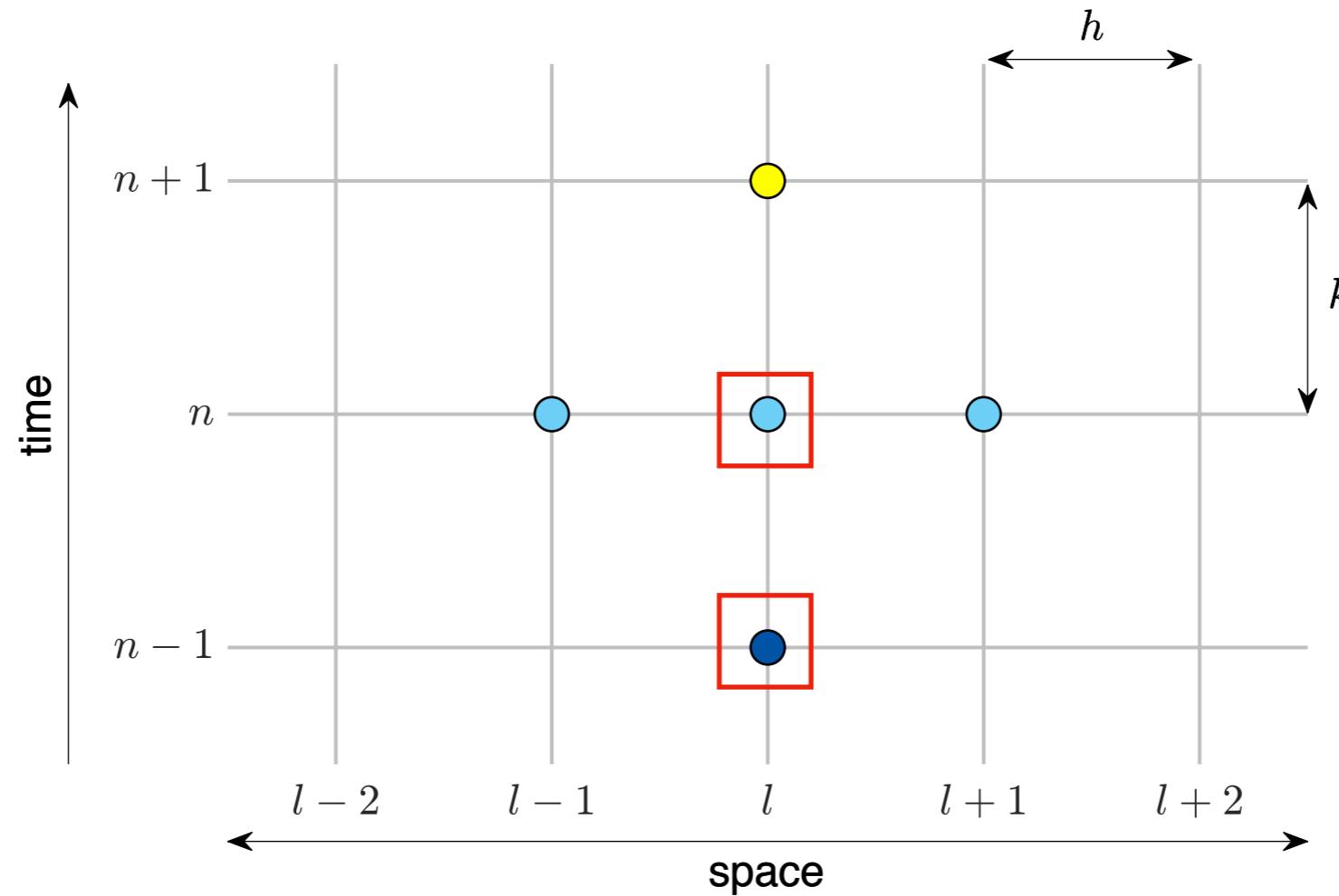
$$u_l^{n+1} = 2u_l^n - u_l^{n-1} + \lambda^2(u_{l+1}^n - 2u_l^n + u_{l-1}^n)$$

Damped 1D wave equation



What about the the stencil of $\delta_{tt}u_l^n = c^2\delta_{xx}u_l^n - 2\sigma_0\delta_{t-}u_l^n$?

$$\dots - 2\sigma_0/k (u_l^n - u_l^{n-1})$$

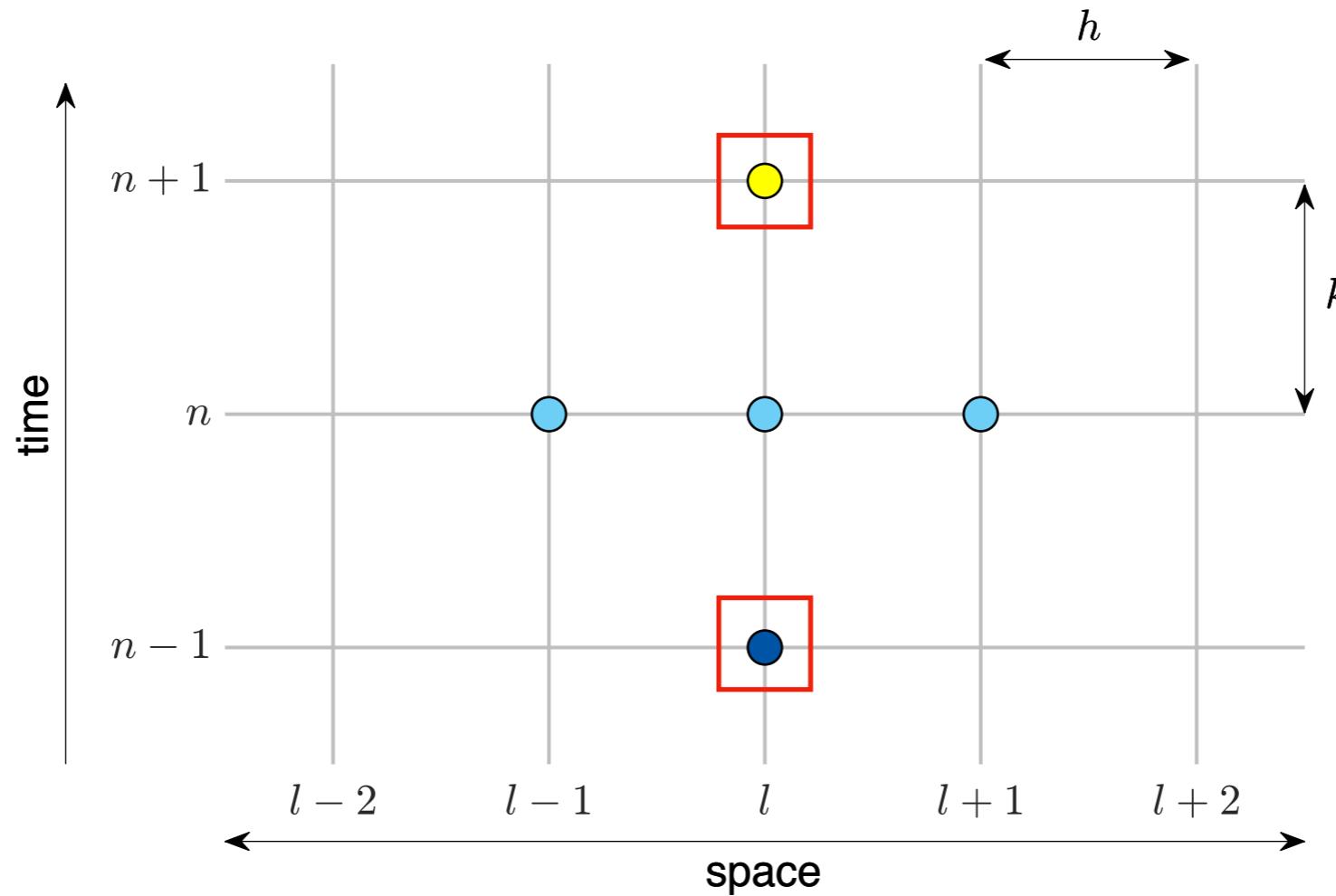


Damped 1D wave equation



Does anything change for $\delta_{tt}u_l^n = c^2\delta_{xx}u_l^n - 2\sigma_0\delta_{t\cdot}u_l^n$?

$$\dots - \sigma_0/k (u_l^{n+1} - u_l^{n-1})$$

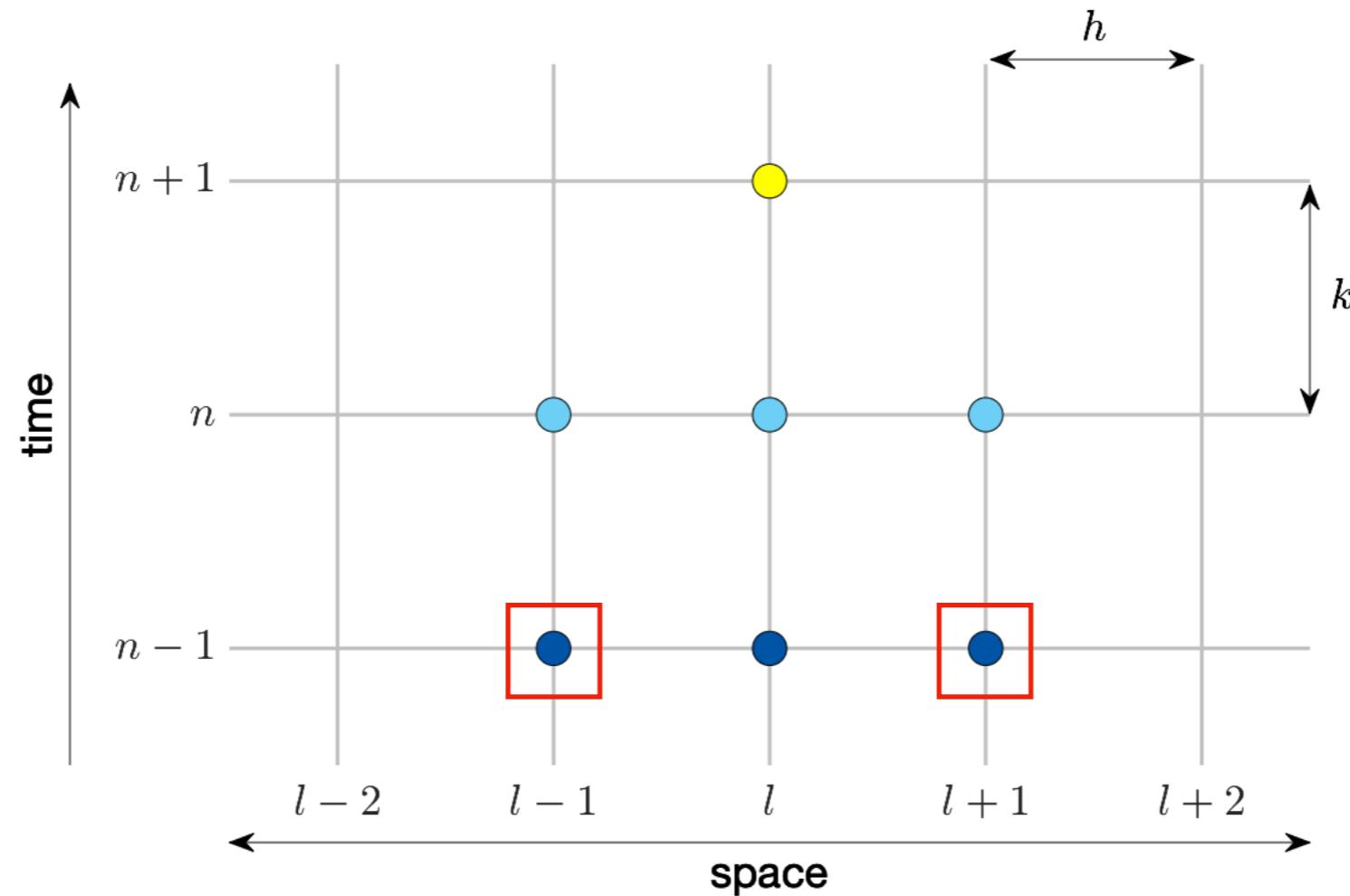




Damped 1D wave equation

What about $\delta_{tt}u_l^n = c^2\delta_{xx}u_l^n - 2\sigma_0\delta_{t+}u_l^n + 2\sigma_1\delta_{t-}\delta_{xx}u_l^n$?

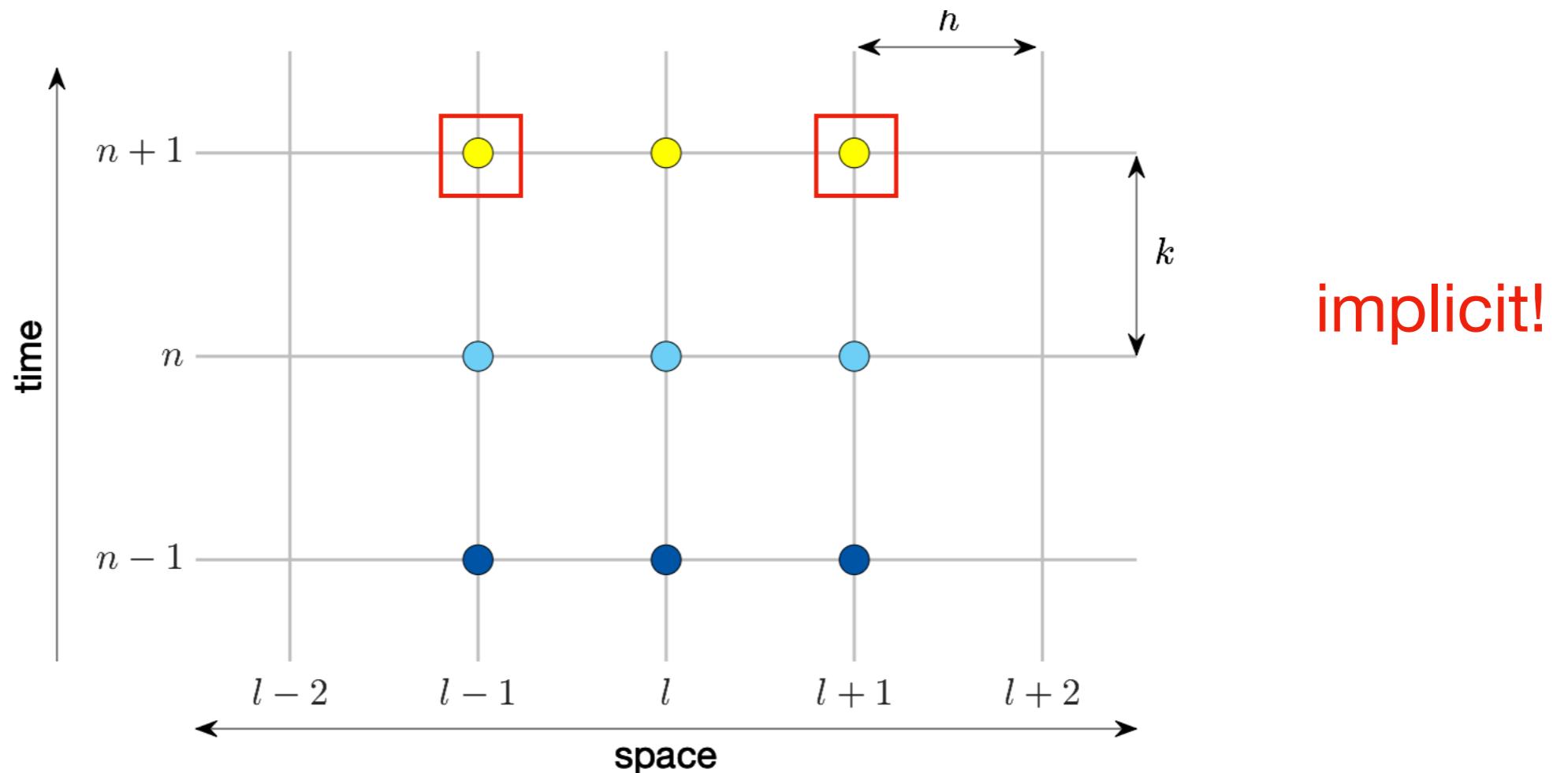
$$\dots + 2\sigma_1/kh^2 (u_{l+1}^n - 2u_l^n + u_{l-1}^n - \boxed{u_{l+1}^{n-1}} + 2u_l^{n-1} - \boxed{u_{l-1}^{n-1}})$$





Damped 1D wave equation

And finally $\delta_{tt}u_l^n = c^2\delta_{xx}u_l^n - 2\sigma_0\delta_{t.}u_l^n + 2\sigma_1\delta_{t.}\delta_{xx}u_l^n$



Ideal Bar



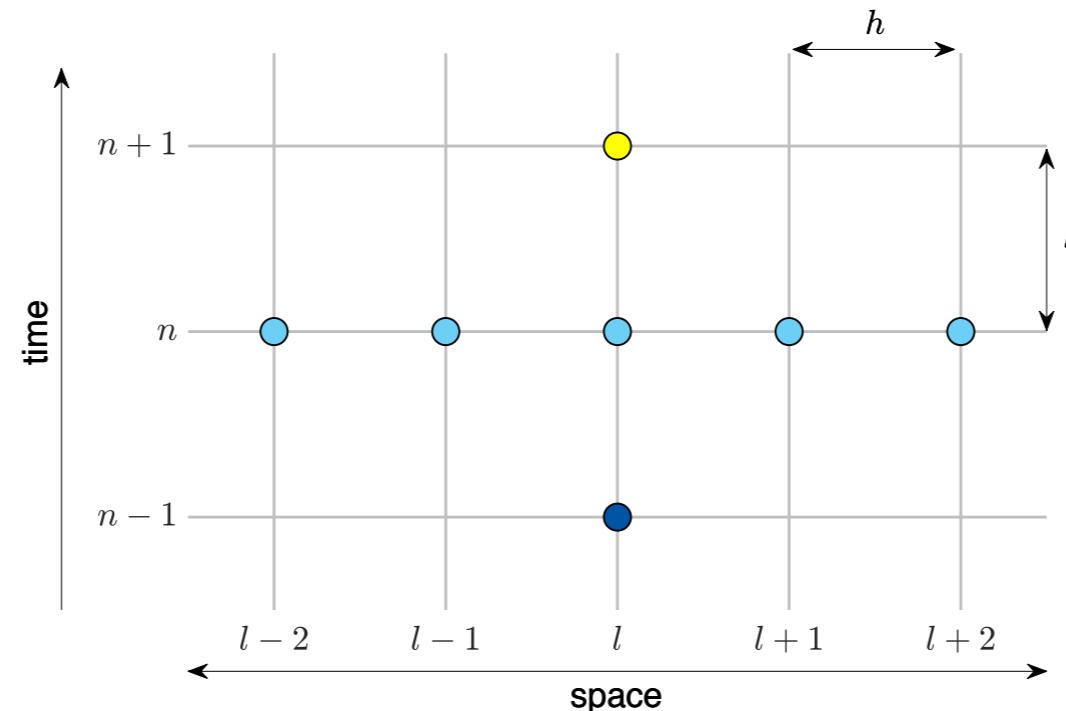
The PDE of the ideal bar is defined as follows:

$$u_{tt} = -\kappa^2 u_{xxxx}$$

with stiffness coefficient κ and 4th-order spatial derivative:

$$u_{xxxx} \approx \delta_{xxxx} u_l^n = \delta_{xx} \delta_{xx} u_l^n = \frac{1}{h^4} (u_{l+2}^n - 4u_{l+1}^n + 6u_l^n - 4u_{l-1}^n + u_{l-2}^n)$$

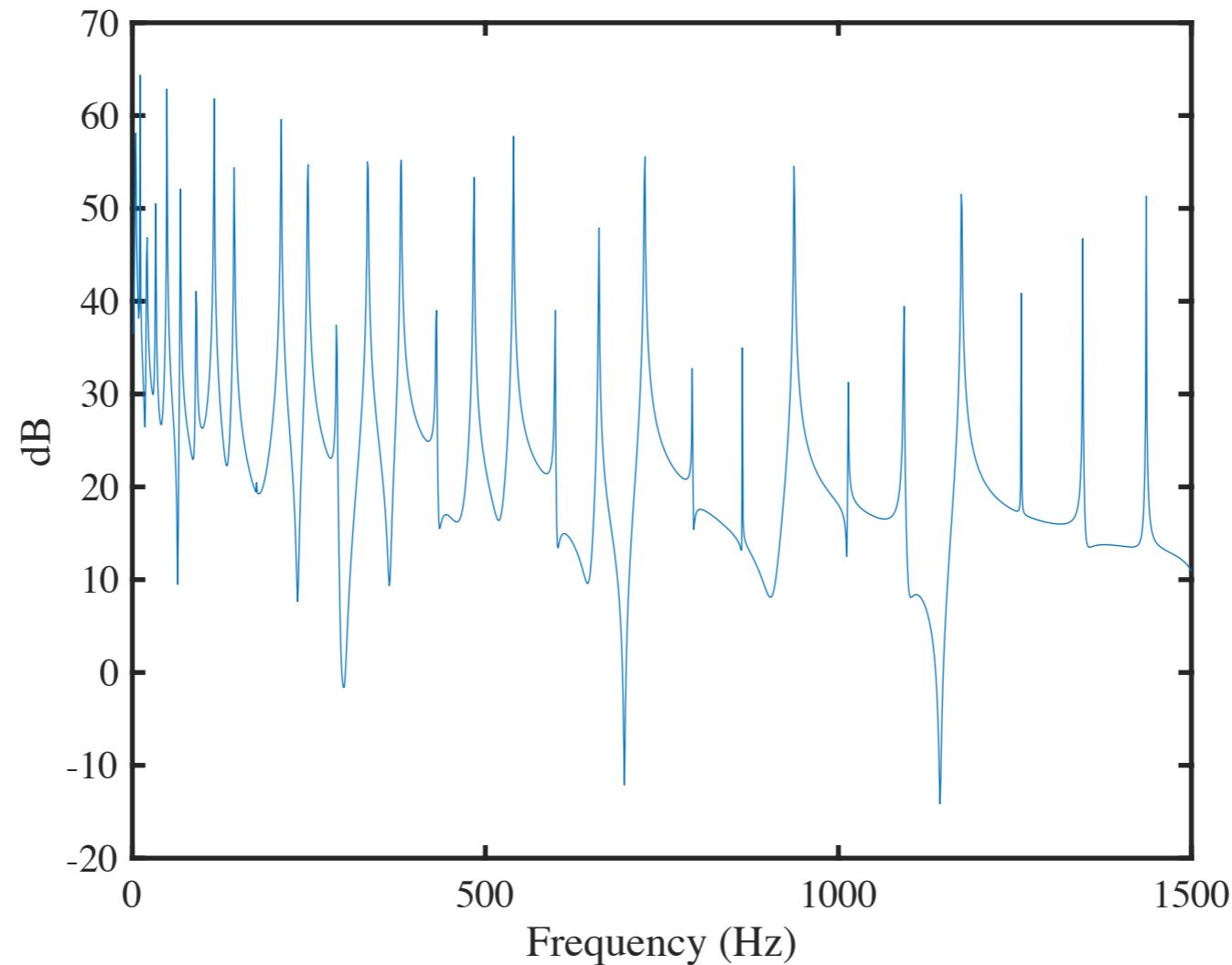
Stencil?





The 4th-order derivative causes **dispersion**:

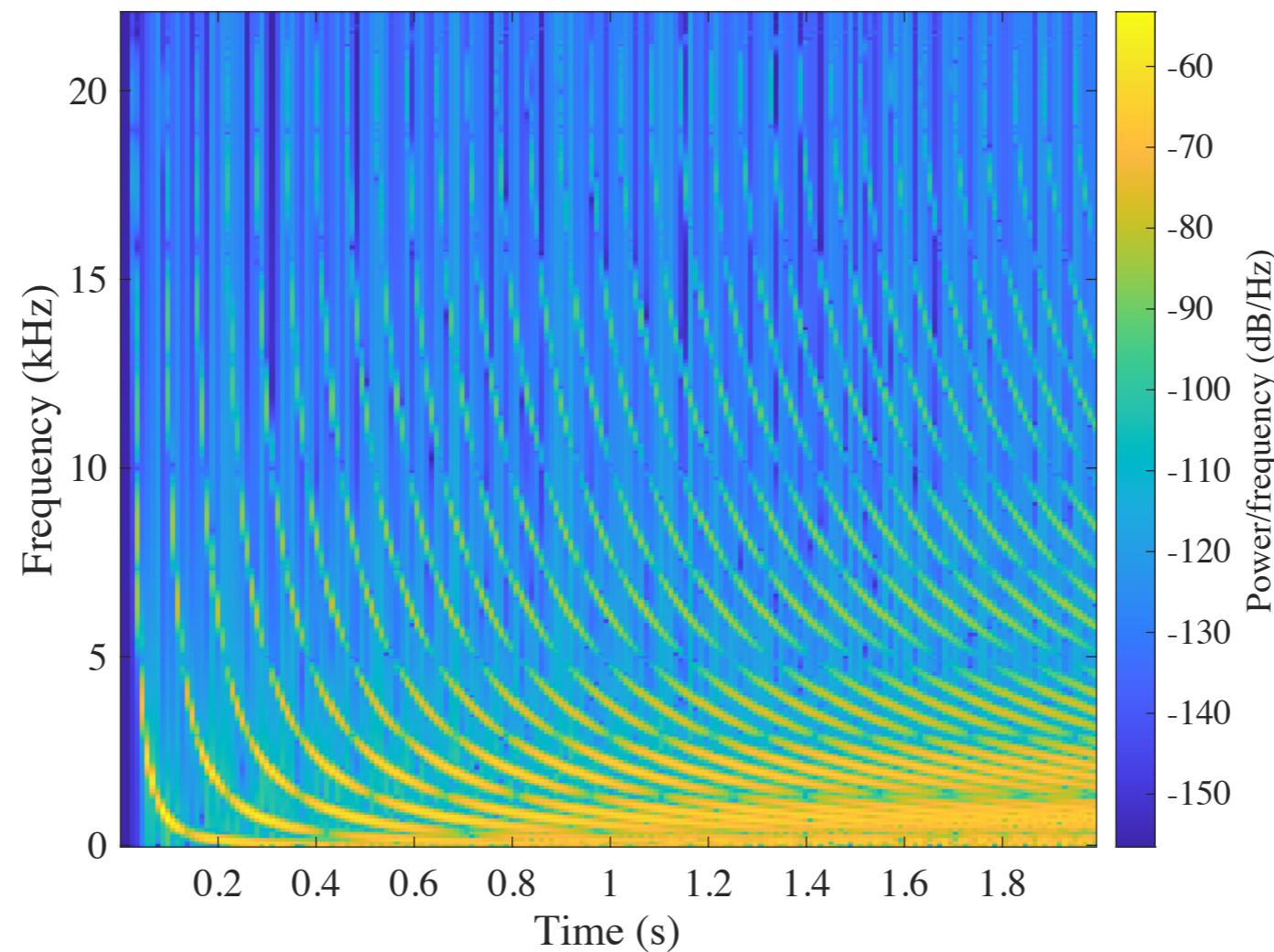
- higher frequencies travel faster than lower ones





Very low values of κ can be used to model a spring reverb

- $\kappa = 0.006, \sigma_0 = 0.1, \sigma_1 = 0.000001$



or star wars
sound effects



For the 1D wave equation, our range of calculation was
 $l = [1, \dots, N - 1]$.

In the case of the bar, we seem to need points outside our defined region! These are referred to as **virtual grid points**.

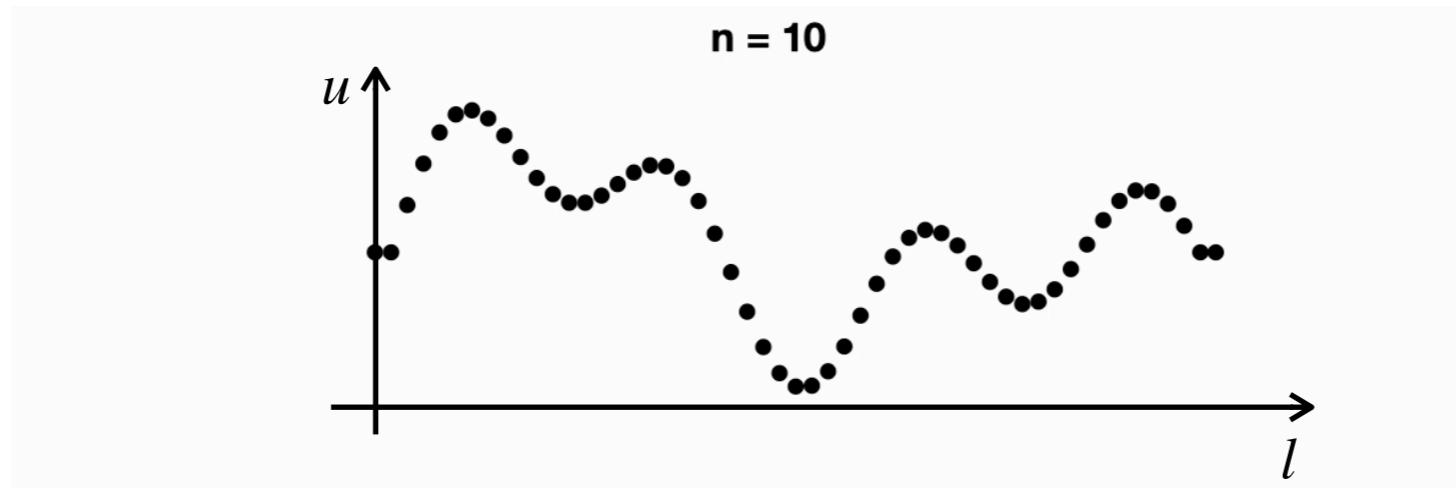
At the boundaries ($l = 0, N$), the following boundary conditions can be defined:

- Clamped $u = u_x = 0$
- Simply supported $u = u_{xx} = 0$
- Free $u_{xx} = u_{xxx} = 0$

From these, we can obtain definitions for the undefined points.

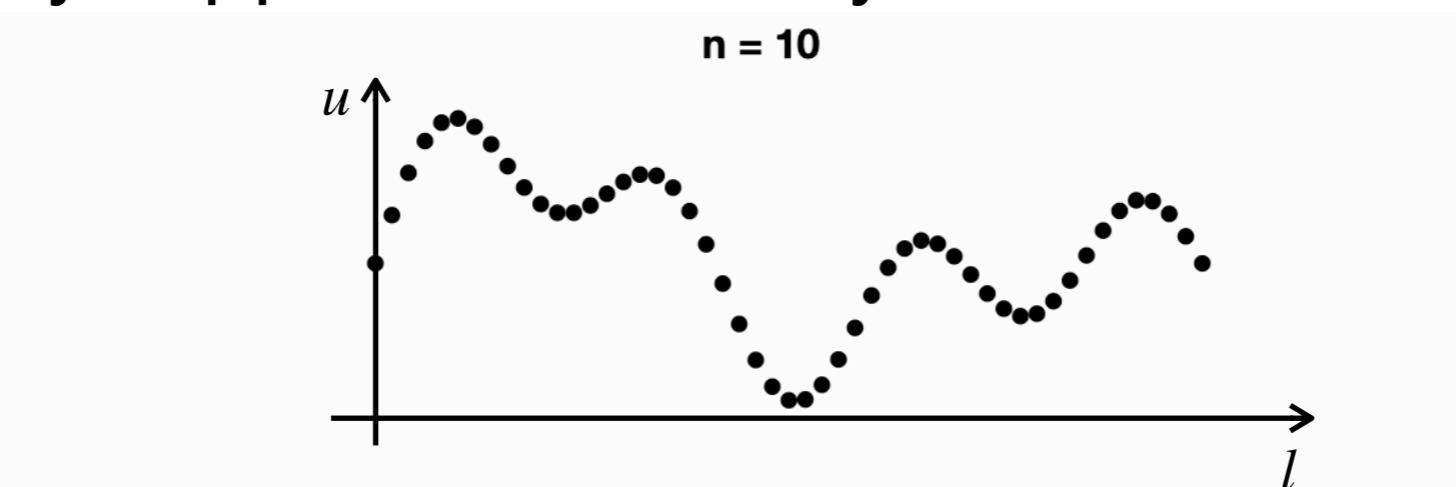


The clamped boundary condition looks like



where we only need to calculate the points $l = [2, \dots, N - 2]$.

The simply supported boundary condition looks like



where we need to calculate the points $l = [1, \dots, N - 1]$.



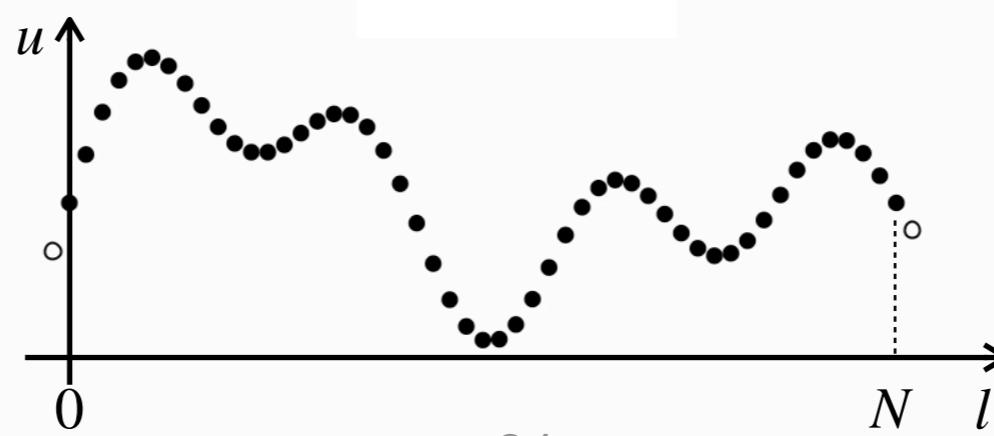
For the simply supported boundary condition, we need to find definitions for virtual grid points when calculating the scheme at $l = 1$ and $l = N - 1$:

$$\dots - \frac{\kappa^2}{h^4} (u_3^n - 4u_2^n + 6u_1^n - 4u_0^n + u_{-1}^n) \quad \dots - \frac{\kappa^2}{h^4} (u_{N+1}^n - 4u_N^n + 6u_{N-1}^n - 4u_{N-2}^n + u_{N-3}^n)$$

We can discretise $u = u_{xx} = 0$ and solve for these points at $l = 0, N$:

$$u_l^n = \boxed{\delta_{xx} u_l^n = 0} \quad \text{expand at } l=0 \Rightarrow \frac{1}{h^2} (u_1^n - 2u_0^n + u_{-1}^n) = 0 \quad u_0^n = 0 \Rightarrow u_{-1}^n = -u_1^n \\ \dots \quad u_{N+1}^n = -u_{N-1}^n$$

Visualised, this looks something like:





So the expansion of $\delta_{xxxx} u_l^n$ at $l = 1$

$$\dots - \frac{\kappa^2}{h^4} (u_3^n - 4u_2^n + 6u_1^n - 4u_0 + u_{-1}^n)$$
$$u_{-1}^n = -u_1^n$$

becomes

$$\dots - \frac{\kappa^2}{h^4} (u_3^n - 4u_2^n + 5u_1^n - 4u_0)$$

The same goes for $l = N - 1$.



Let's implement the ideal bar using the simply supported boundary condition.

Advice: start with clamped ($l = [2, \dots, N - 2]$)

The (unexpanded) FDS is:

$$\delta_{tt} u_l^n = -\kappa^2 \delta_{xxxx} u_l^n$$

with

$$\delta_{xxxx} u_l^n = \frac{1}{h^4} (u_{l+2}^n - 4u_{l+1}^n + 6u_l^n - 4u_{l-1}^n + u_{l-2}^n)$$

Boundary condition: $u_l^n = \delta_{xx} u_l^n = 0$ at $l = 0, N$

For stability, use: $\implies u_{-1}^n = -u_1^n$ and $u_{N+1}^n = -u_{N-1}^n$

$$h = \sqrt{2\kappa k}$$

Stiff String

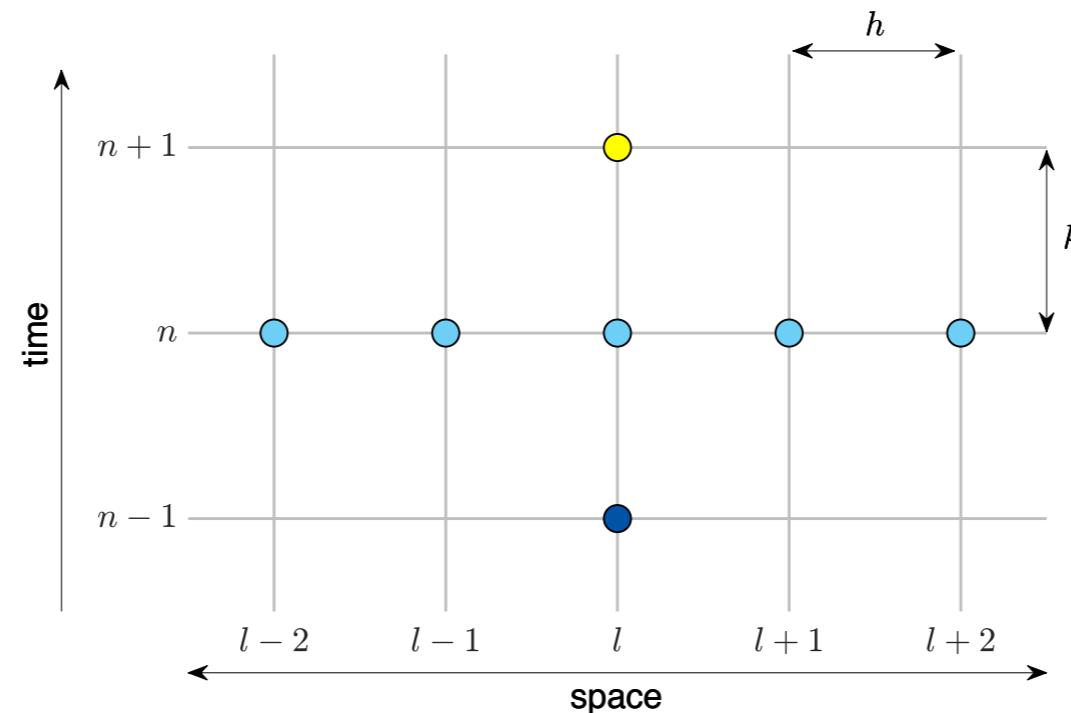


Stiff String

Basically a combination of the 1D wave equation and the ideal bar.

$$u_{tt} = c^2 u_{xx} - \kappa^2 u_{xxxx}$$

Stencil:



For stability, use:

$$h = \sqrt{\frac{c^2 k^2 + \sqrt{c^4 k^4 + 16\kappa^2 k^2}}{2}}$$

Damped Stiff String

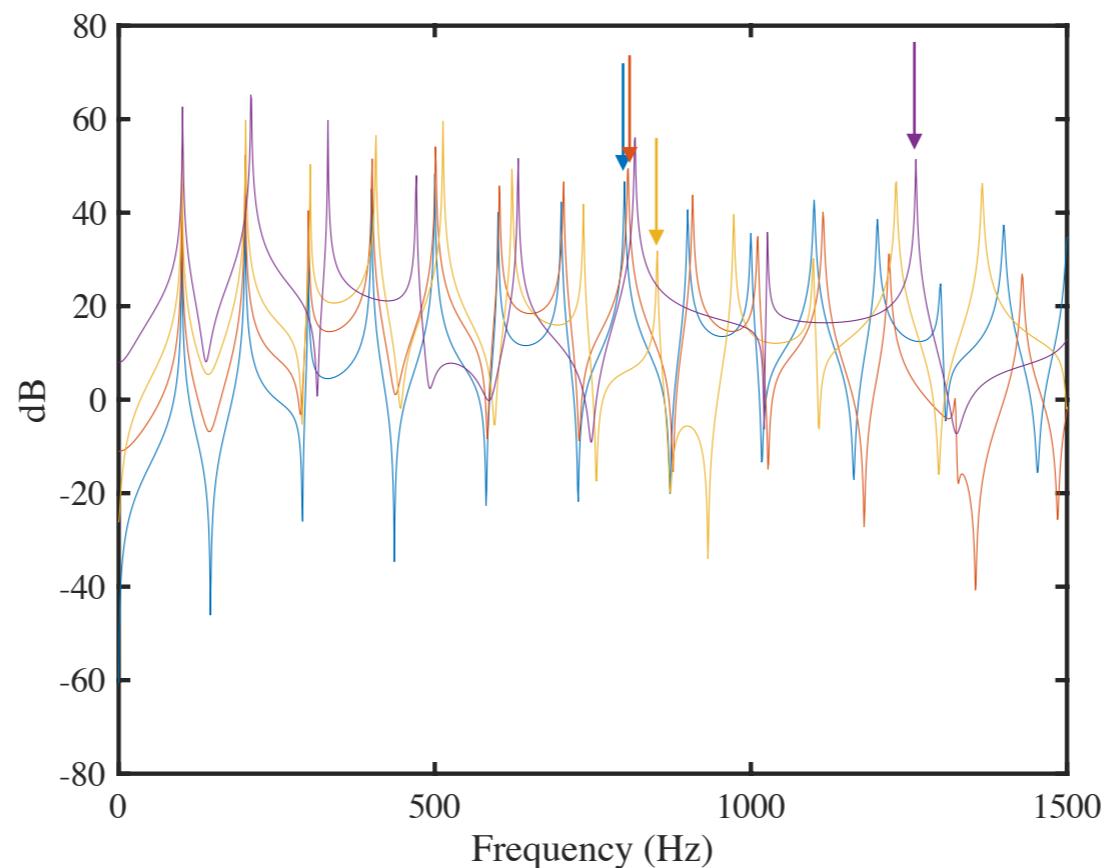


Damped Stiff String

We can (again) simply add terms to the PDE:

$$u_{tt} = c^2 u_{xx} - \kappa^2 u_{xxxx} - 2\sigma_0 u_t + 2\sigma_1 u_{txx}$$

Sound examples: $c = 200$, $L = 1$, $\sigma_0 = 1$, $\sigma_1 = 0.005$, $\kappa = 10$



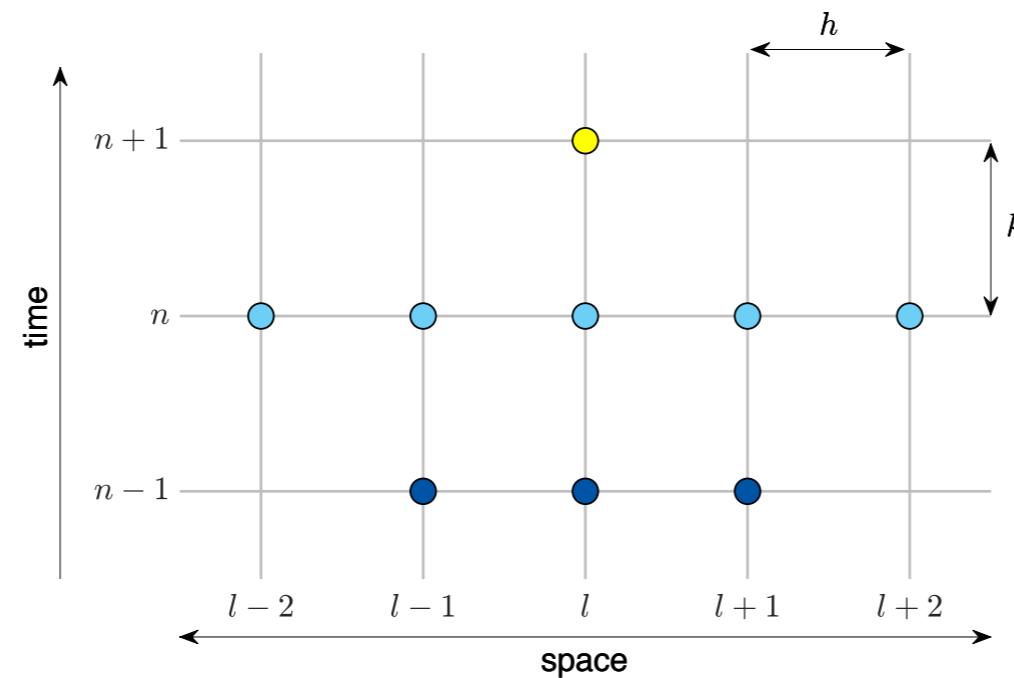


Damped Stiff String

(Explicit) FDS is:

$$\delta_{tt} u_l^n = c^2 \delta_{xx} u_l^n - \kappa^2 \delta_{xxxx} u_l^n - 2\sigma_0 \delta_t u_l^n + 2\sigma_1 \delta_{t-} \delta_{xx} u_l^n$$

Stencil?

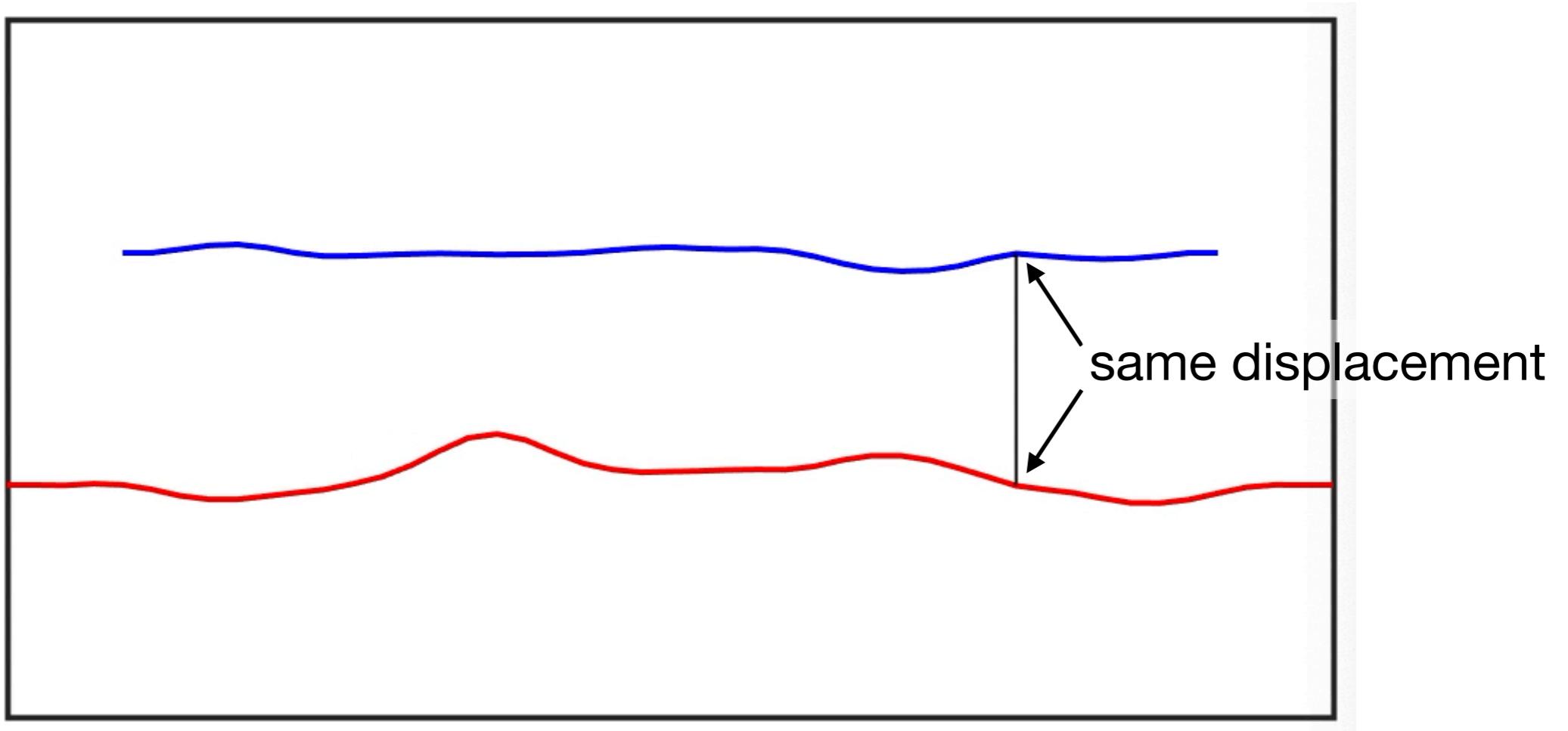


Stability: $h = \sqrt{\frac{c^2 k^2 + 4\sigma_1 k + \sqrt{(c^2 k^2 + 4\sigma_1 k)^2 + 16\kappa^2 k^2}}{2}}$

Towards complete
instruments



We can connect different models





When connecting different models, it is important to write out all physical parameters.

- Undamped ideal bar:

$$\rho A u_{tt} = -EI u_{xxxx}$$

- Stiffness parameter $\kappa = \sqrt{EI/\rho A}$

A and I can be calculated from radius r (for bar with circular cross-section): $A = \pi r^2$ and $I = \pi r^4/4$.

Symbol	Description	Unit
ρ	material density	kg/m^3
A	cross-sectional area	m^2
E	Young's modulus	Pa
I	moment of inertia	m^4



We can then add some connection force f :

$$\rho A u_{tt} = -EIu_{xxxx} + \delta(x - x_c)f$$

where $\delta(x) = \begin{cases} 1 & x = 0 \\ 0 & \text{otherwise} \end{cases}$ applies force f to location $x_c \in \mathcal{D}$.

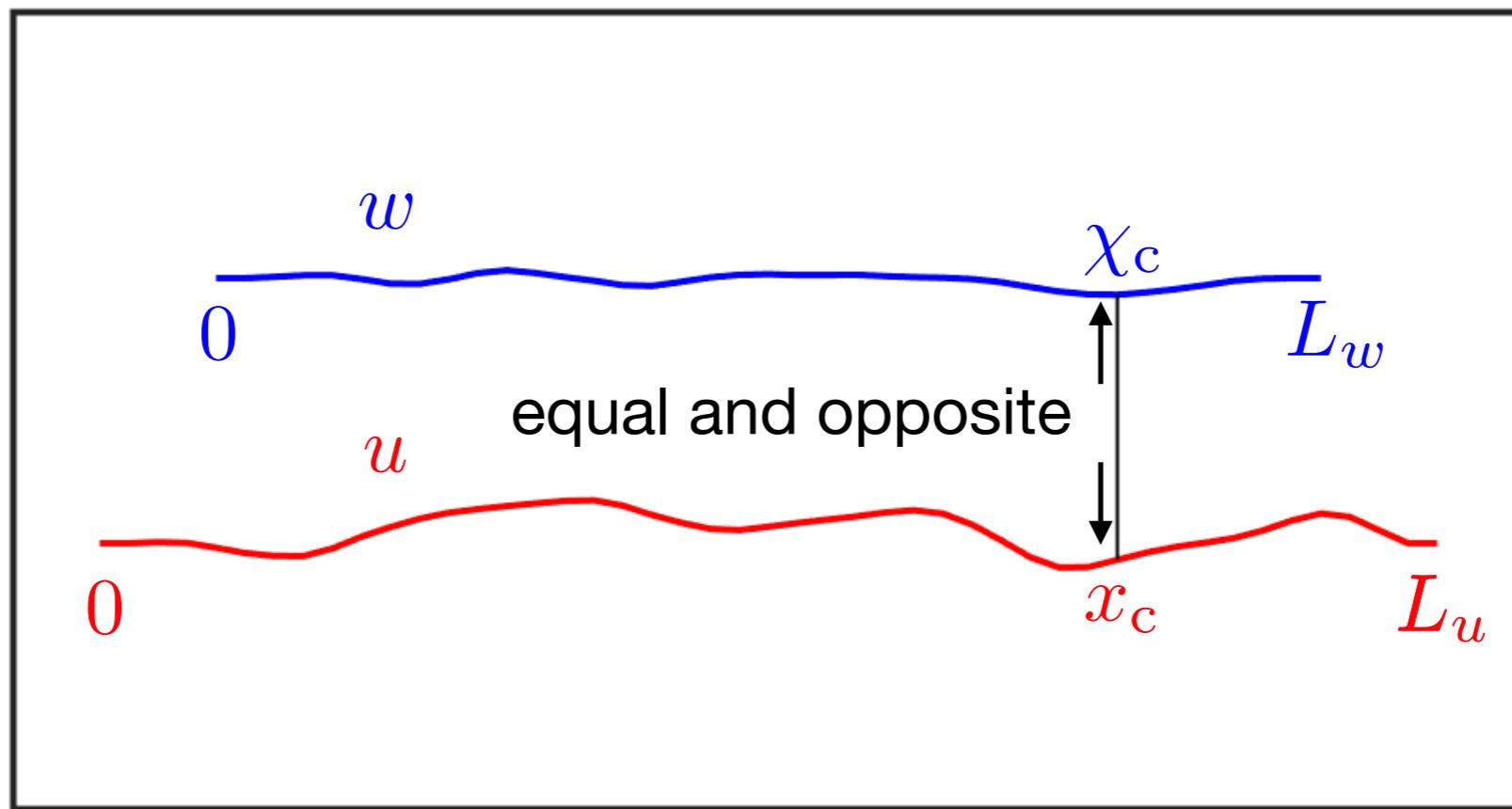




Towards complete instruments

Introducing a second bar $w = w(\chi, t)$:

$$\begin{cases} \rho_1 A_1 u_{tt} = -E_1 I_1 u_{xxxx} + \delta(x - x_c) f \\ \rho_2 A_2 w_{tt} = -E_2 I_2 w_{\chi\chi\chi\chi} \square \delta(\chi - \chi_c) f \end{cases}$$





Introducing a second bar $w = w(\chi, t)$:

$$\begin{cases} \rho_1 A_1 u_{tt} = -E_1 I_1 u_{xxxx} + \delta(x - x_c) f \\ \rho_2 A_2 w_{tt} = -E_2 I_2 w_{\chi\chi\chi\chi} - \delta(\chi - \chi_c) f \end{cases}$$

We can discretise the system as follows:

$$\begin{cases} \rho_1 A_1 \delta_{tt} u_l^n = -E_1 I_1 \delta_{xxxx} u_l^n + J(x_c) f \\ \rho_2 A_2 \delta_{tt} w_l^n = -E_2 I_2 \delta_{\chi\chi\chi\chi} w_l^n - J(\chi_c) f \end{cases}$$

where spreading operator

$$J(x_i) = \begin{cases} \frac{1}{h} & l = l_i = \text{round}(x_i/h) \\ 0 & \text{otherwise} \end{cases}$$

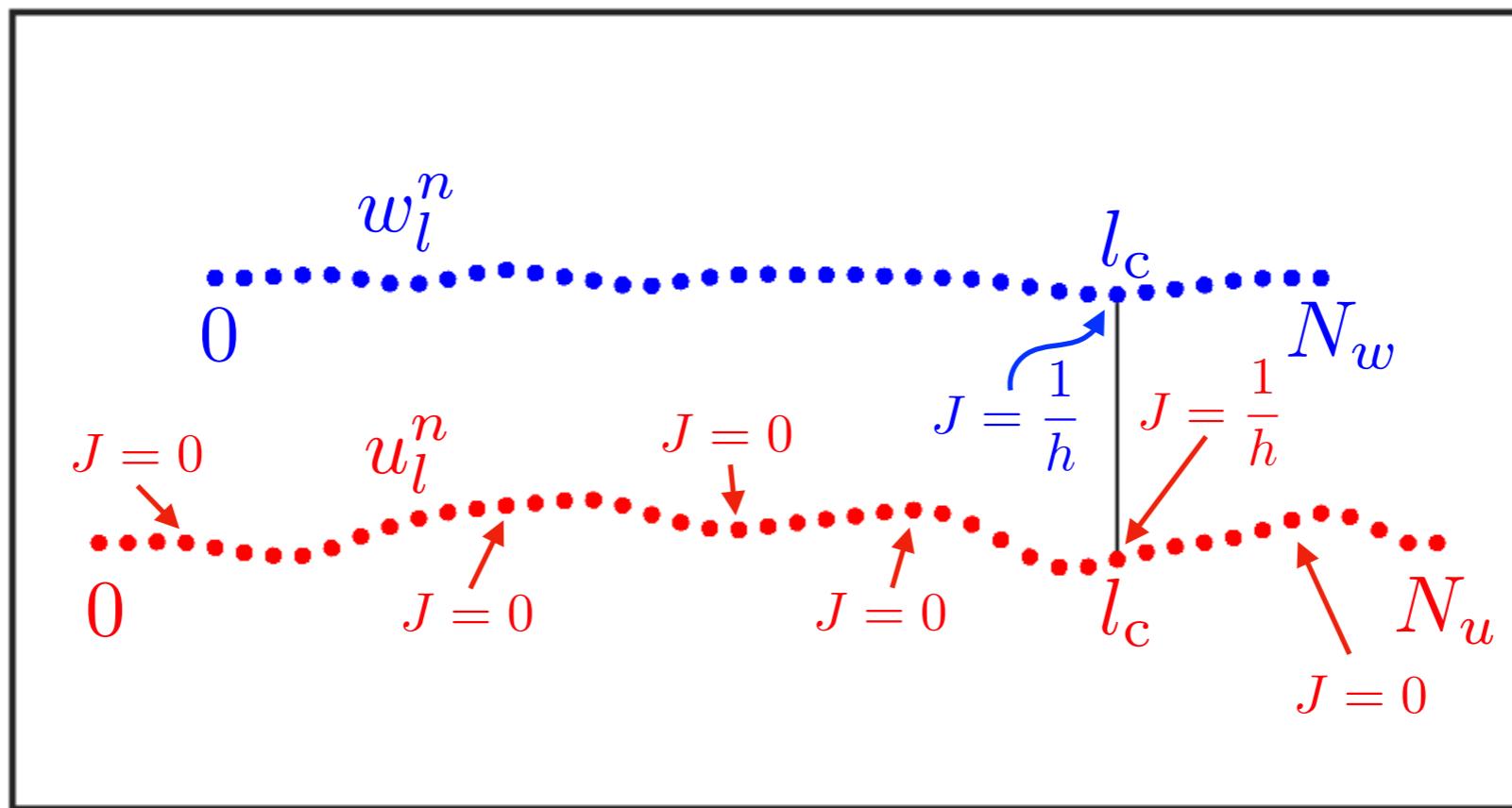
is the discrete version of $\delta(x_i)$.

Towards complete instruments



We can discretise the system as follows:

$$\begin{cases} \rho_1 A_1 \delta_{tt} u_l^n = -E_1 I_1 \delta_{xxxx} u_l^n + J(x_c) f \\ \rho_2 A_2 \delta_{tt} w_l^n = -E_2 I_2 \delta_{\chi\chi\chi\chi} w_l^n - J(\chi_c) f \end{cases}$$





Towards complete instruments

We can discretise the system as follows:

$$\begin{cases} \rho_1 A_1 \delta_{tt} u_l^n = -E_1 I_1 \delta_{xxxx} u_l^n + J(x_c) f \\ \rho_2 A_2 \delta_{tt} w_l^n = -E_2 I_2 \delta_{\chi\chi\chi\chi} w_l^n - J(\chi_c) f \end{cases}$$

Now we need to solve for f .

The first step is to divide by ρA

$$\begin{cases} \delta_{tt} u_l^n = -\kappa_1^2 \delta_{xxxx} u_l^n + J(x_c) \frac{f}{\rho_1 A_1} \\ \delta_{tt} w_l^n = -\kappa_2^2 \delta_{\chi\chi\chi\chi} w_l^n - J(\chi_c) \frac{f}{\rho_2 A_2} \end{cases}$$

importance of using physical parameters

Then, we look at the schemes at their connection locations

$$\begin{cases} \delta_{tt} u_{l_c}^n = -\kappa_1^2 \delta_{xxxx} u_{l_c}^n + \frac{f}{\rho_1 A_1 h_1} \\ \delta_{tt} w_{l_c}^n = -\kappa_2^2 \delta_{\chi\chi\chi\chi} w_{l_c}^n - \frac{f}{\rho_2 A_2 h_2} \end{cases}$$

weight of one grid point

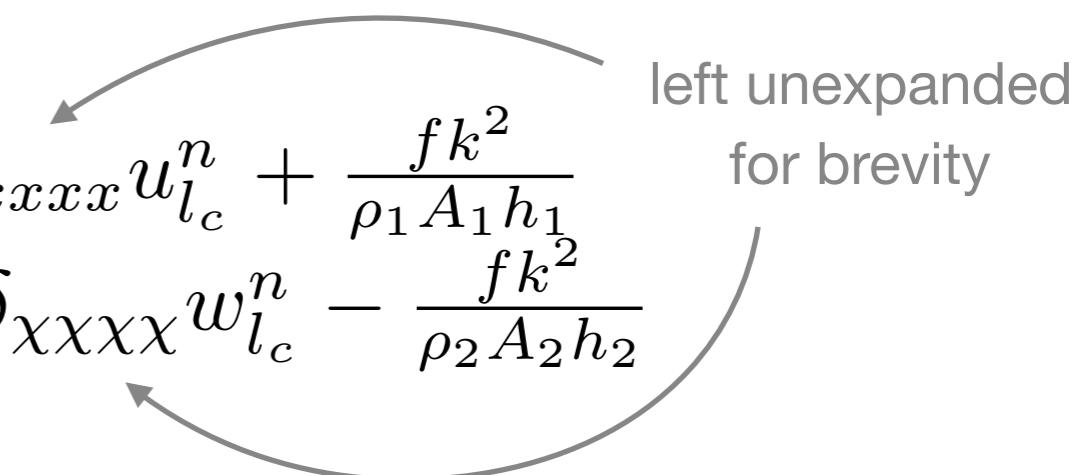


Then, we look at the schemes at their connection locations

$$\begin{cases} \delta_{tt} u_{l_c}^n = -\kappa_1^2 \delta_{xxxx} u_{l_c}^n + \frac{f}{\rho_1 A_1 h_1} \\ \delta_{tt} w_{l_c}^n = -\kappa_2^2 \delta_{\chi\chi\chi\chi} w_{l_c}^n - \frac{f}{\rho_2 A_2 h_2} \end{cases}$$

expand, and solve for $u_{l_c}^{n+1}$ and $w_{l_c}^{n+1}$

$$\begin{cases} u_{l_c}^{n+1} = 2u_{l_c}^n - u_{l_c}^{n-1} - \kappa_1^2 k^2 \delta_{xxxx} u_{l_c}^n + \frac{fk^2}{\rho_1 A_1 h_1} \\ w_{l_c}^{n+1} = 2w_{l_c}^n - w_{l_c}^{n-1} - \kappa_2^2 k^2 \delta_{\chi\chi\chi\chi} w_{l_c}^n - \frac{fk^2}{\rho_2 A_2 h_2} \end{cases}$$



How we proceed next depends on the connection type

- Rigid: $u_{l_c}^n = w_{l_c}^n$ for all n
- Spring-like: $f = -K \mu_t \cdot \eta^n$ with $\eta^n = u_{l_c}^n - w_{l_c}^n$

Towards complete instruments



$$\begin{cases} u_{l_c}^{n+1} = 2u_{l_c}^n - u_{l_c}^{n-1} - \kappa_1^2 k^2 \delta_{xxxx} u_{l_c}^n + \frac{fk^2}{\rho_1 A_1 h_1} \\ w_{l_c}^{n+1} = 2w_{l_c}^n - w_{l_c}^{n-1} - \kappa_2^2 k^2 \delta_{\chi\chi\chi\chi} w_{l_c}^n - \frac{fk^2}{\rho_2 A_2 h_2} \end{cases}$$

↗ = ↘

Using a rigid connection, $u_{l_c}^n = w_{l_c}^n$ for all n , we also know that $u_{l_c}^{n+1} = w_{l_c}^{n+1}$

$$2u_{l_c}^n - u_{l_c}^{n-1} - \kappa^2 k^2 \delta_{xxxx} u_{l_c}^n + \frac{fk^2}{\rho_1 A_1 h_1} = 2w_{l_c}^n - w_{l_c}^{n-1} - \kappa^2 k^2 \delta_{\chi\chi\chi\chi} w_{l_c}^n - \frac{fk^2}{\rho_2 A_2 h_2}$$

$$\left(\frac{k^2}{\rho_1 A_1 h_1} + \frac{k^2}{\rho_2 A_2 h_2} \right) f = -\kappa^2 k^2 \delta_{\chi\chi\chi\chi} w_{l_c}^n + \kappa^2 k^2 \delta_{xxxx} u_{l_c}^n$$

$$f = \frac{-\kappa_2^2 k^2 \delta_{\chi\chi\chi\chi} w_{l_c}^n + \kappa_1^2 k^2 \delta_{xxxx} u_{l_c}^n}{\frac{k^2}{\rho_1 A_1 h_1} + \frac{k^2}{\rho_2 A_2 h_2}}$$



Implementation

1. Calculate schemes without connection forces (the usual)

intermediate

$$\begin{cases} u_l^I = 2u_l^n - u_l^{n-1} - \kappa_1^2 k^2 \delta_{xxxx} u_l^n \\ w_l^I = 2w_l^n - w_l^{n-1} - \kappa_2^2 k^2 \delta_{\chi\chi\chi\chi} w_l^n \end{cases}$$

uNext(range) = ...
wNext(range) = ...

where $u_{l_c}^I + \frac{fk^2}{\rho_1 A_1 h_1} = u_{l_c}^{n+1}$ and $w_{l_c}^I - \frac{fk^2}{\rho_2 A_2 h_2} = w_{l_c}^{n+1}$.

2. Calculate force using intermediate states

$$u_{l_c}^I + \frac{fk^2}{\rho_1 A_1 h_1} = w_{l_c}^I - \frac{fk^2}{\rho_2 A_2 h_2} \implies f = \frac{w_{l_c}^I - u_{l_c}^I}{\left(\frac{k^2}{\rho_1 A_1 h_1} + \frac{k^2}{\rho_2 A_2 h_2} \right)}$$

force = ...

3. Add force to intermediate states to get states at $n + 1$

$$u_{l_c}^{n+1} = u_{l_c}^I + \frac{fk^2}{\rho_1 A_1 h_1} \quad w_{l_c}^{n+1} = w_{l_c}^I - \frac{fk^2}{\rho_2 A_2 h_2}$$

uNext(lcu) =
uNext(lcu) + ...
wNext(lcw) =
wNext(lcw) + ...