

# Controlled Quadrotor ODEs



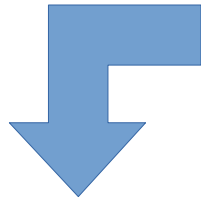
**Aircraft Dynamics**  
**UNIVERSITY OF COLORADO BOULDER**

# Last time: Linearization

Hover Trim Condition

+

Taylor Series  
Expansion



$$\begin{pmatrix} \Delta \dot{x}_E \\ \Delta \dot{y}_E \\ \Delta \dot{z}_E \end{pmatrix} = \begin{pmatrix} \Delta u \\ \Delta v \\ \Delta w \end{pmatrix}$$

$$\begin{pmatrix} \Delta \dot{\phi} \\ \Delta \dot{\theta} \\ \Delta \dot{\psi} \end{pmatrix} = \begin{pmatrix} \Delta p \\ \Delta q \\ \Delta r \end{pmatrix}$$

$$\begin{pmatrix} \Delta \dot{u} \\ \Delta \dot{v} \\ \Delta \dot{w} \end{pmatrix} = g \begin{pmatrix} -\Delta \theta \\ \Delta \phi \\ 0 \end{pmatrix} + \frac{1}{m} \begin{pmatrix} 0 \\ 0 \\ \Delta Z_c \end{pmatrix}$$

$$\begin{pmatrix} \Delta \dot{p} \\ \Delta \dot{q} \\ \Delta \dot{r} \end{pmatrix} = \begin{pmatrix} \frac{1}{I_x} \Delta L_c \\ \frac{1}{I_y} \Delta M_c \\ \frac{1}{I_z} \Delta N_c \end{pmatrix}$$



$$\begin{pmatrix} \dot{x}_E \\ \dot{y}_E \\ \dot{z}_E \end{pmatrix} = \begin{pmatrix} c_\theta c_\psi & s_\phi s_\theta c_\psi - c_\phi s_\psi & c_\phi s_\theta c_\psi + s_\phi s_\psi \\ c_\theta s_\psi & s_\phi s_\theta s_\psi + c_\phi c_\psi & c_\phi s_\theta s_\psi - s_\phi c_\psi \\ -s_\theta & s_\phi c_\theta & c_\phi c_\theta \end{pmatrix} \begin{pmatrix} u^E \\ v^E \\ w^E \end{pmatrix}$$

$$\begin{pmatrix} \dot{\phi} \\ \dot{\theta} \\ \dot{\psi} \end{pmatrix} = \begin{pmatrix} 1 & \sin \phi \tan \theta & \cos \phi \tan \theta \\ 0 & \cos \phi & -\sin \phi \\ 0 & \sin \phi \sec \theta & \cos \phi \sec \theta \end{pmatrix} \begin{pmatrix} p \\ q \\ r \end{pmatrix}$$

$$\begin{pmatrix} \dot{u}^E \\ \dot{v}^E \\ \dot{w}^E \end{pmatrix} = \begin{pmatrix} rv^E - qw^E \\ pw^E - ru^E \\ qu^E - pv^E \end{pmatrix} + g \begin{pmatrix} -\sin \theta \\ \cos \theta \sin \phi \\ \cos \theta \cos \phi \end{pmatrix} + \frac{1}{m} \begin{pmatrix} X \\ Y \\ Z \end{pmatrix} + \frac{1}{m} \begin{pmatrix} 0 \\ 0 \\ Z_c \end{pmatrix}$$

$$\begin{pmatrix} \dot{p} \\ \dot{q} \\ \dot{r} \end{pmatrix} = \begin{pmatrix} \frac{I_y - I_z}{I_x} qr \\ \frac{I_z - I_x}{I_y} pr \\ \frac{I_x - I_y}{I_z} pq \end{pmatrix} + \begin{pmatrix} \frac{1}{I_x} L \\ \frac{1}{I_y} M \\ \frac{1}{I_z} N \end{pmatrix} + \begin{pmatrix} \frac{1}{I_x} L_c \\ \frac{1}{I_y} M_c \\ \frac{1}{I_z} N_c \end{pmatrix}$$

Nonlinear Quadrotor  
Equations of Motion

**Longitudinal**

$$\begin{pmatrix} \Delta \dot{x}_E \\ \Delta \dot{u} \\ \Delta \dot{\theta} \\ \Delta \dot{q} \end{pmatrix} = \begin{pmatrix} \Delta u \\ -g \Delta \theta \\ \Delta q \\ \frac{1}{I_u} \Delta M_c \end{pmatrix}$$

**Lateral**

$$\begin{pmatrix} \Delta \dot{y}_E \\ \Delta \dot{v} \\ \Delta \dot{\phi} \\ \Delta \dot{p} \end{pmatrix} = \begin{pmatrix} \Delta v \\ g \Delta \phi \\ \Delta p \\ \frac{1}{I_r} \Delta L_c \end{pmatrix}$$

**Vertical**

$$\begin{pmatrix} \Delta \dot{z}_E \\ \Delta \dot{w} \end{pmatrix} = \begin{pmatrix} \Delta w \\ \frac{1}{m} \Delta Z_c \end{pmatrix}$$

**Spin**

$$\begin{pmatrix} \Delta \dot{\psi} \\ \Delta \dot{r} \end{pmatrix} = \begin{pmatrix} \Delta r \\ \frac{1}{I_z} \Delta N_c \end{pmatrix}$$



# Linearized Quadrotor EOM

## Longitudinal

$$\begin{pmatrix} \Delta \dot{x}_E \\ \Delta \dot{u} \\ \Delta \dot{\theta} \\ \Delta \dot{q} \end{pmatrix} = \begin{pmatrix} \Delta u \\ -g \Delta \theta \\ \Delta q \\ \frac{1}{I_u} \Delta M_c \end{pmatrix}$$

## Lateral

$$\begin{pmatrix} \Delta \dot{y}_E \\ \Delta \dot{v} \\ \Delta \dot{\phi} \\ \Delta \dot{p} \end{pmatrix} = \begin{pmatrix} \Delta v \\ q \Delta \phi \\ \Delta p \\ \frac{1}{I_x} \Delta L_c \end{pmatrix}$$

Today: Solution of 2<sup>nd</sup> order linear ODEs

## Vertical

$$\begin{pmatrix} \Delta \dot{z}_E \\ \Delta \dot{w} \end{pmatrix} = \begin{pmatrix} \Delta w \\ \frac{1}{m} \Delta Z_c \end{pmatrix}$$

## Spin

$$\begin{pmatrix} \Delta \dot{\psi} \\ \Delta \dot{r} \end{pmatrix} = \begin{pmatrix} \Delta r \\ \frac{1}{I_z} \Delta N_c \end{pmatrix}$$



# Linear ODEs

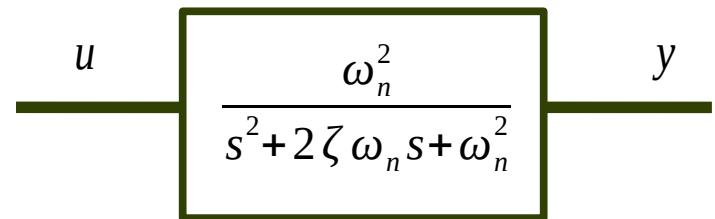
$$\dot{x} = A x + B u$$
$$y = C x + D u$$

Aircraft dynamics (mostly)  
uses this one  
It is called a “state space”  
model.

These contain (roughly) the  
same information!

Always keep this in  
mind!

$$\ddot{y} + 2\zeta \omega_n \dot{y} + \omega_n^2 y = \omega_n^2 u$$



# Roll Angle State Space Model

$$\begin{pmatrix} \Delta \dot{\phi} \\ \Delta \dot{p} \end{pmatrix} = \begin{pmatrix} \Delta p \\ \frac{1}{I_x} \Delta L_c \end{pmatrix}$$

$$\begin{pmatrix} \Delta \dot{\phi} \\ \Delta \dot{p} \end{pmatrix} = \begin{pmatrix} 0 & 1 \\ 0 & 0 \end{pmatrix} \begin{pmatrix} \Delta \phi \\ \Delta p \end{pmatrix} + \begin{pmatrix} 0 \\ 1/I_x \end{pmatrix} \Delta L_c$$

$$\dot{x} = A x + B u$$

$$\phi = \begin{pmatrix} 1 & 0 \end{pmatrix} \begin{pmatrix} \Delta \phi \\ \Delta p \end{pmatrix} + \begin{pmatrix} 0 \end{pmatrix} \Delta L_c$$

$$y = C x + D u$$



# Open Loop Solution

$$\begin{pmatrix} \Delta \dot{\phi} \\ \Delta \dot{p} \end{pmatrix} = \begin{pmatrix} \Delta p \\ \frac{1}{I_x} \Delta L_c \end{pmatrix}$$

$$\Delta \ddot{\phi} = \Delta \dot{p}$$

$$\Delta \ddot{\phi} = \frac{1}{I_x} \Delta L_c$$

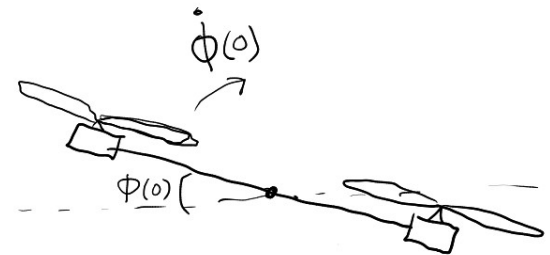
What are the solutions?

Depends on  $\Delta L_c$

$\Delta L_c = 0$  (hover  
 $\Rightarrow \Delta \ddot{\phi} = 0$  control)

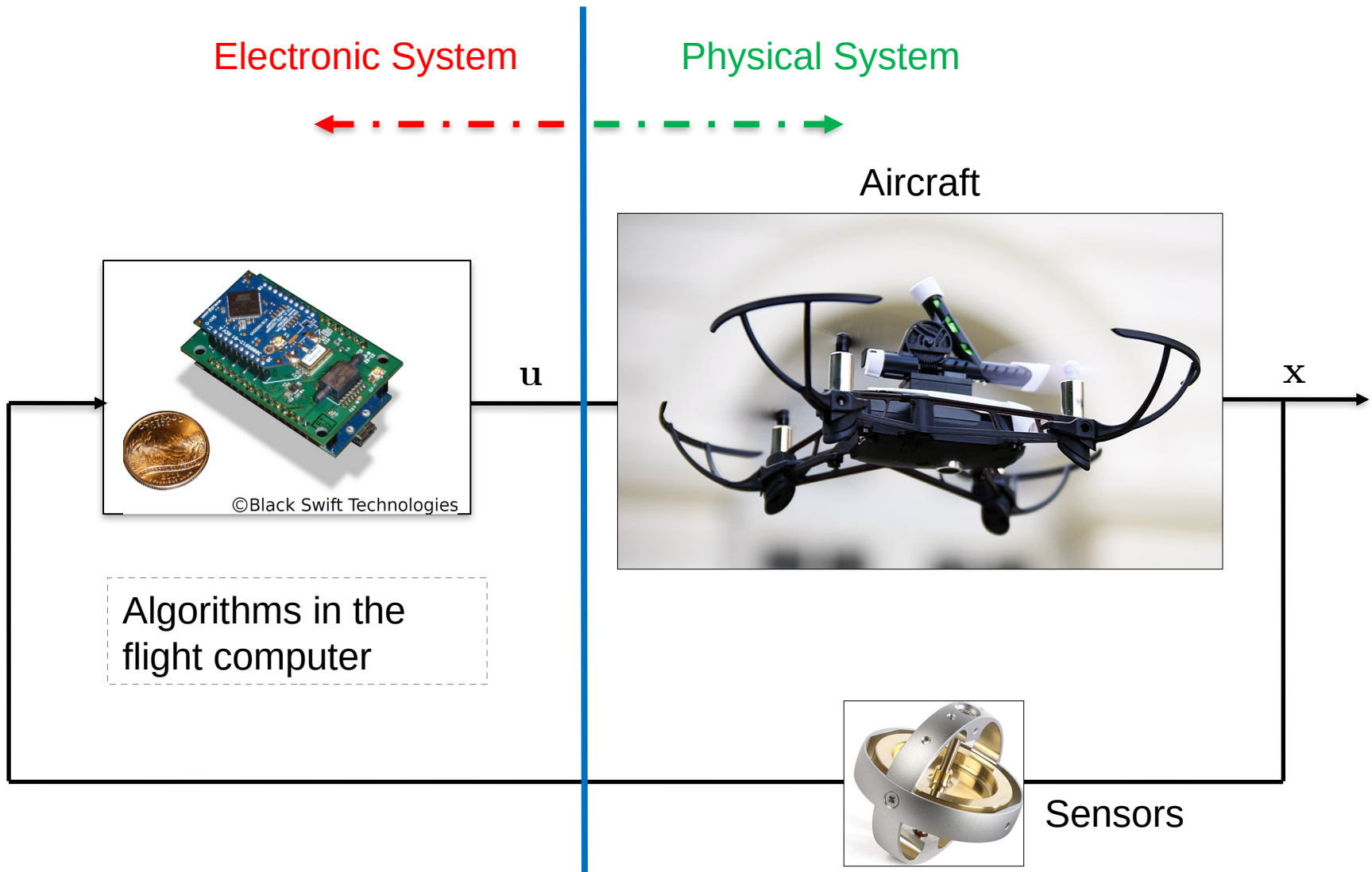
$$\Delta \dot{\phi}(t) = \int \Delta \ddot{\phi} = 0 + \Delta \dot{\phi}(0)$$

$$\Delta \phi(t) = \int \Delta \dot{\phi} = \Delta \dot{\phi}(0)t + \Delta \phi(0)$$

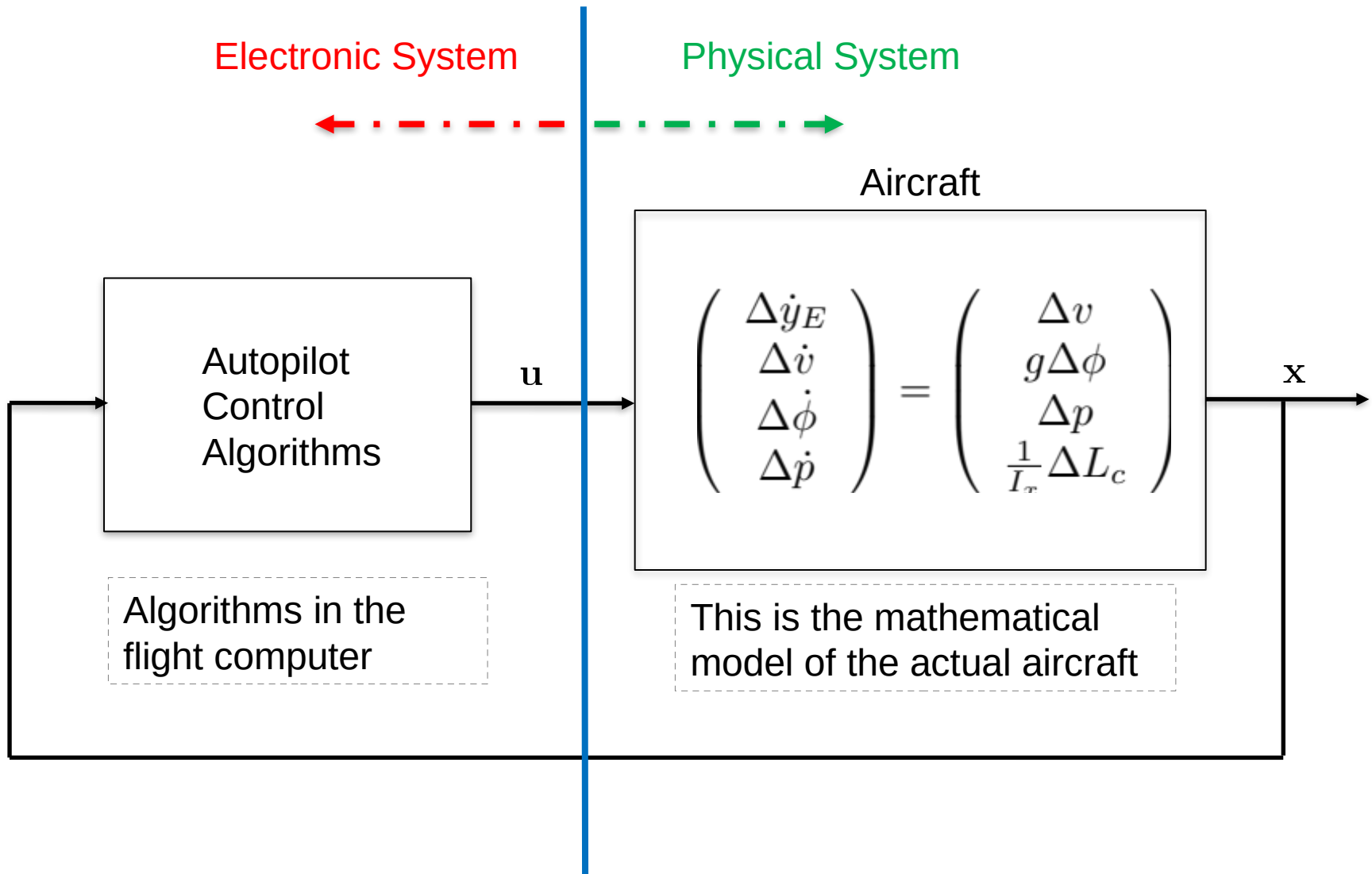


If initial conditions are nonzero, quad will crash!

# Feedback Control

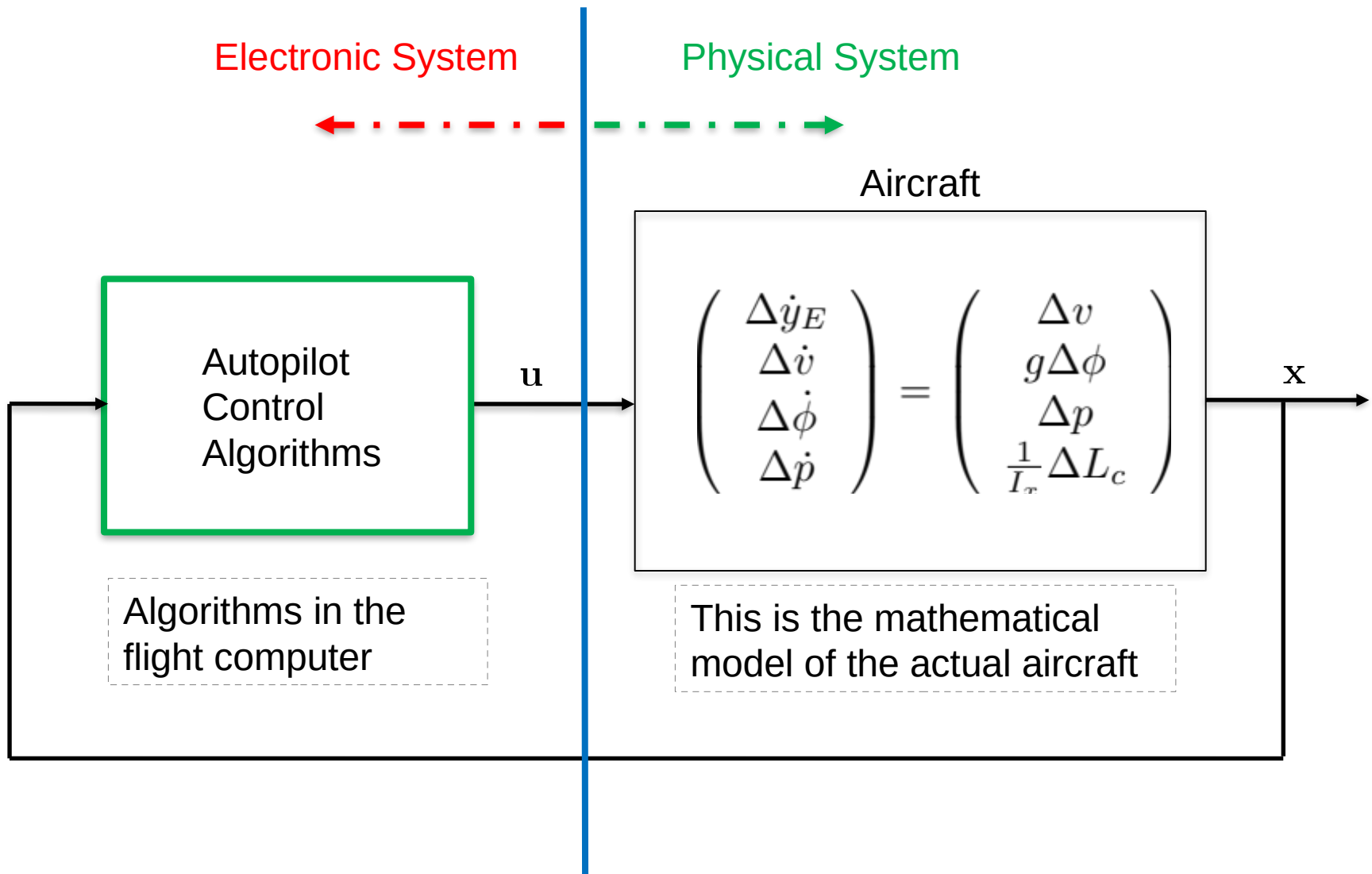


# Feedback Control

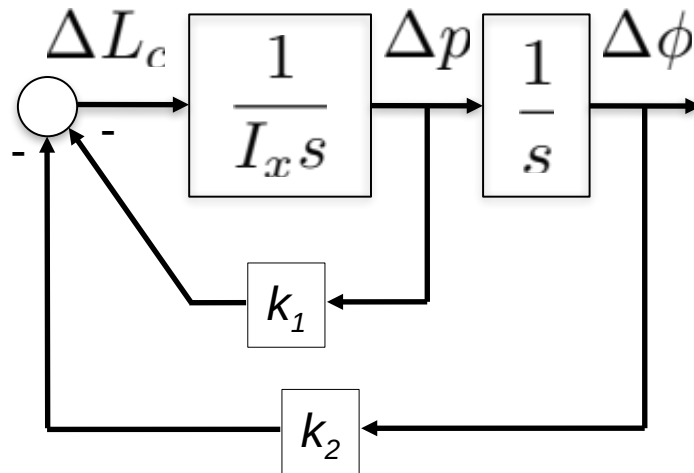




# Feedback Control



# Control Strategy for Roll Angle



$$\Delta L_c = -k_1 \Delta p - k_2 \Delta \phi$$

Derivative

Proportional

$$\dot{x} = Ax + Bu$$

$$u = -Kx = -(k_1 \quad k_2) \begin{pmatrix} \Delta p \\ \Delta \phi \end{pmatrix}$$

$$\begin{aligned} \dot{x} &= Ax - BKx \\ &= (A - BK)x \end{aligned}$$

$$= A^{cl} x$$

$A^{cl} = A - BK$  is the **closed loop dynamics matrix**

$$\dot{x} = A^{cl} x$$

is the **closed loop state space model** with controller integrated

# Closed Loop Behavior

There are two ways mathematically to describe the resulting “closed loop behavior”

$$\begin{pmatrix} \Delta \dot{\phi} \\ \Delta \dot{p} \end{pmatrix} = \begin{pmatrix} \Delta p \\ \frac{1}{I_x} \Delta L_c \end{pmatrix}$$

$$\Delta \ddot{\phi} = \Delta \dot{p} = \frac{1}{I_x} \Delta L_c$$

$$\Delta \ddot{\phi} = \frac{1}{I_x} (-k_1 \Delta p - k_2 \Delta \phi)$$

$$\Delta \ddot{\phi} + \frac{k_1}{I_x} \Delta \dot{\phi} + \frac{k_2}{I_x} \Delta \phi = 0$$

$$\begin{pmatrix} \Delta \dot{\phi} \\ \Delta \dot{p} \end{pmatrix} = \begin{pmatrix} \Delta p \\ \frac{1}{I_x} \Delta L_c \end{pmatrix}$$

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$$\begin{pmatrix} \Delta \dot{\phi} \\ \Delta \dot{p} \end{pmatrix} = \begin{pmatrix} 0 & 1 \\ \frac{-k_2}{I_x} & \frac{-k_1}{I_x} \end{pmatrix} \begin{pmatrix} \Delta \phi \\ \Delta p \end{pmatrix}$$

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# Closed Loop Behavior

2<sup>nd</sup> Order System perspective

$$\Delta\ddot{\phi} + \frac{k_1}{I_x}\Delta\dot{\phi} + \frac{k_2}{I_x}\Delta\phi = 0$$



$$\Delta\ddot{\phi} + 2\zeta\omega_n\Delta\dot{\phi} + \omega_n^2\Delta\phi = 0$$

$$\omega_n = \sqrt{\frac{k_2}{I_x}} \quad \zeta = \frac{k_1}{2\sqrt{k_2 I_x}}$$

which has roots  $\lambda = -\zeta\omega_n \pm i\omega_n\sqrt{1 - \zeta^2}$

The response to an initial condition is determined by the natural frequency and damping ratio.



# Review: Poles of a system

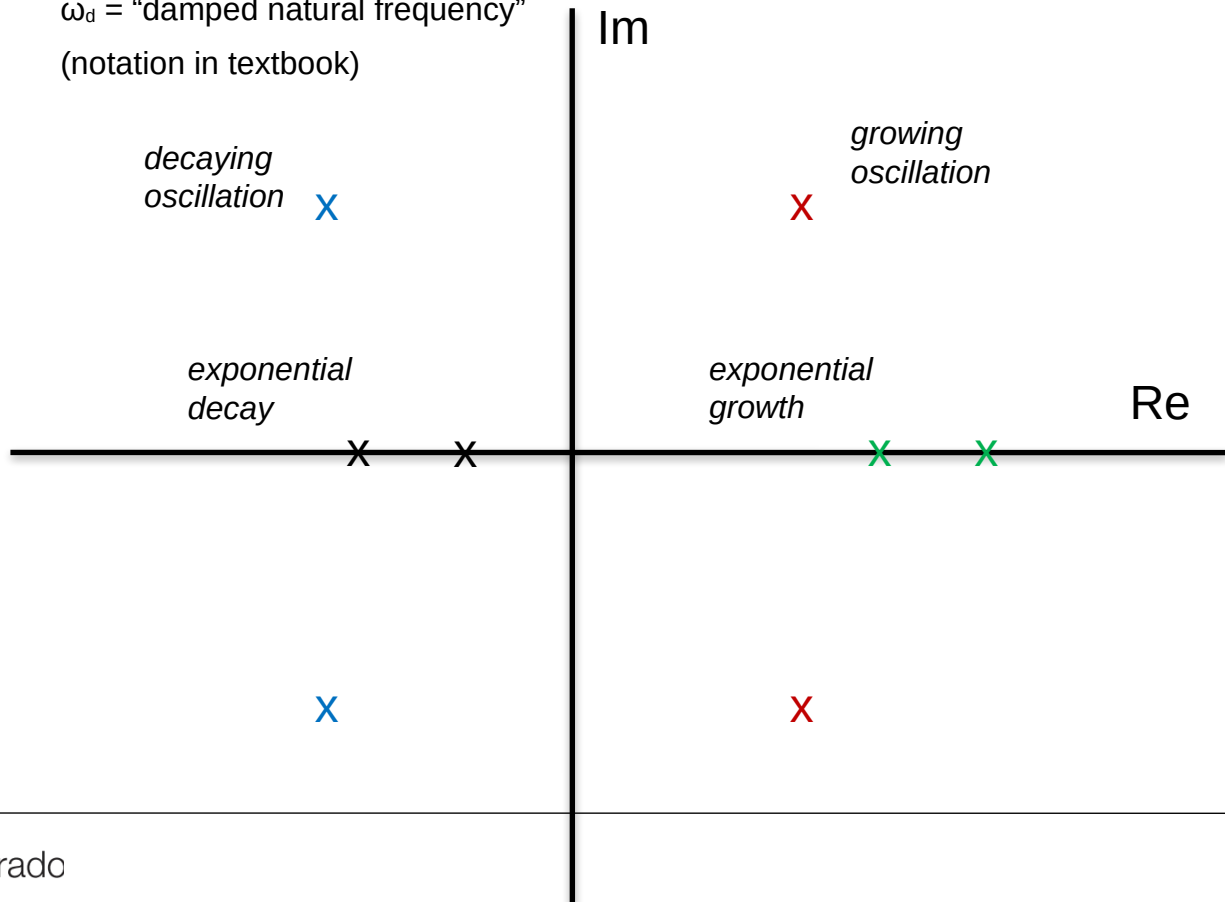
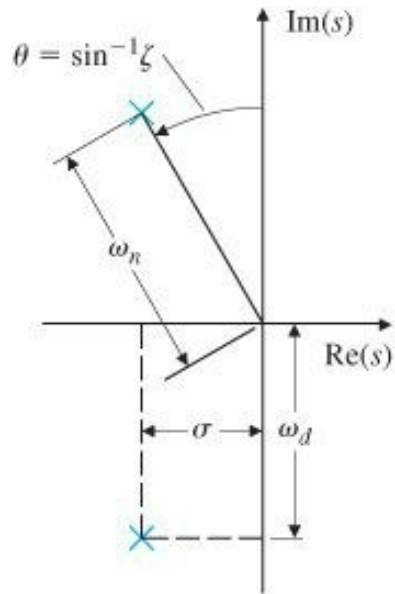
The roots of the characteristic equation are called the “poles” of the system and determine the dynamic stability properties

$$\lambda = -\zeta\omega_n \pm i\omega_n\sqrt{1-\zeta^2}$$

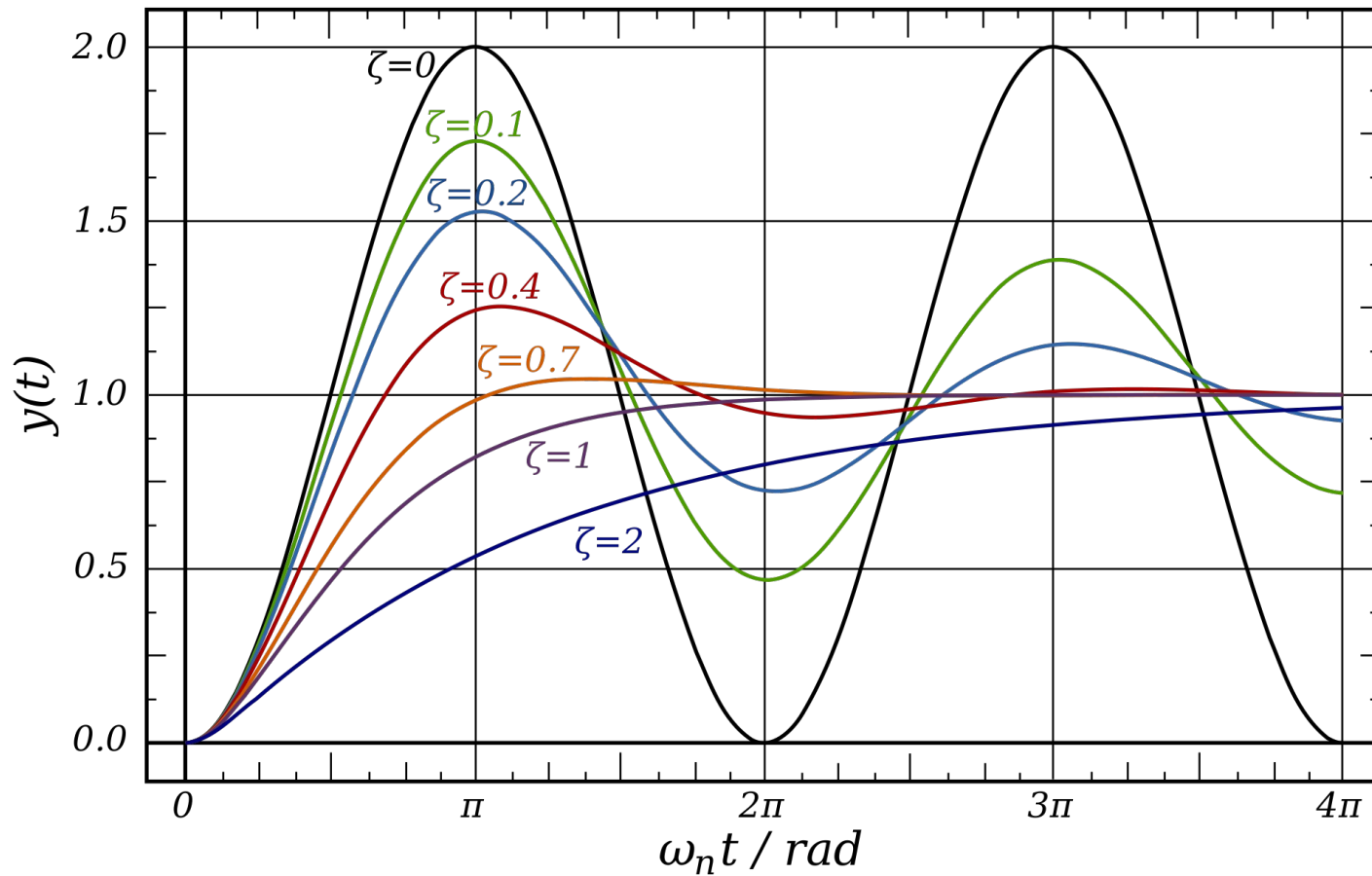
$$\lambda = -\sigma \pm \omega_d i$$

$$\lambda = n \pm \omega i$$

$\omega_d$  = “damped natural frequency”  
(notation in textbook)



# Review: Damping Ratio



# Closed Loop Behavior

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$$\Delta \ddot{\phi} = \frac{1}{I_x} (-k_1 \Delta p - k_2 \Delta \phi)$$

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This closed loop state space model can be analyzed with tools from linear algebra



# Break



# Closed Loop Behavior

There are two ways mathematically to describe the resulting “closed loop behavior”

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This “state space model” can be analyzed with tools from linear algebra

# Linear System Natural Response

$$\dot{\mathbf{x}} = \mathbf{A}\mathbf{x} + \mathbf{B}\mathbf{u}$$

Consider natural response

$$\dot{\mathbf{x}} = \mathbf{A}\mathbf{x}$$

Take the Laplace transform

$$s\mathbf{X}(s) - \mathbf{x}(0) = \mathbf{A}\mathbf{X}(s)$$

$$(s\mathbf{I} - \mathbf{A})\mathbf{X}(s) = \mathbf{x}(0)$$

$$\mathbf{X}(s) = (s\mathbf{I} - \mathbf{A})^{-1} \mathbf{x}(0)$$

Taylor series expansion

$$\mathbf{X}(s) = \left( \frac{\mathbf{I}}{s} + \frac{\mathbf{A}}{s^2} + \frac{\mathbf{A}^2}{s^3} \dots \right) \mathbf{x}(0)$$

Take the inverse Laplace transform

$$\mathbf{x}(t) = \left( \mathbf{I} + t\mathbf{A} + \frac{t^2}{2!}\mathbf{A}^2 \dots \right) \mathbf{x}(0)$$

This looks familiar...

$$e^{at} = 1 + ta + \frac{t^2}{2!}a^2 + \frac{t^3}{3!}a^3 \dots$$

We make the definition (need to be careful not to infer too many properties)

$$e^{\mathbf{A}t} = \mathbf{I} + t\mathbf{A} + \frac{t^2}{2!}\mathbf{A}^2 \dots$$

So we have

$$\boxed{\mathbf{x}(t) = e^{\mathbf{A}t} \mathbf{x}(0)}$$

# Modal Analysis

Recall eigenvectors and eigenvalue

$$\mathbf{A}\mathbf{v}_i = \lambda_i\mathbf{v}_i$$

Consider the initial condition vector as one of the eigenvalues

$$\begin{aligned}\mathbf{x}(t) &= \left( \mathbf{I} + t\mathbf{A} + \frac{t^2}{2!}\mathbf{A}^2 \dots \right) \mathbf{x}(0) \\ &= \left( \mathbf{I} + t\mathbf{A} + \frac{t^2}{2!}\mathbf{A}^2 \dots \right) \mathbf{v}_i \\ &= \left( \mathbf{v}_i + t\mathbf{A}\mathbf{v}_i + \frac{t^2}{2!}\mathbf{A}^2\mathbf{v}_i \dots \right) \\ &= \left( \mathbf{v}_i + t\lambda_i\mathbf{v}_i + \frac{t^2}{2!}\lambda_i^2\mathbf{v}_i \dots \right) \\ &= \left( 1 + t\lambda_i + \frac{t^2}{2!}\lambda_i^2 \dots \right) \mathbf{v}_i = e^{\lambda_i t} \mathbf{v}_i\end{aligned}$$

When initial condition is

$$\mathbf{x}(0) = \sum_{i=1}^n k_i \mathbf{v}_i$$

The solution is

$$\mathbf{x}(t) = \sum_{i=1}^n k_i e^{\lambda_i \cdot t} \mathbf{v}_i$$

This is a critical fact! For eigenvector, matrix exponential solution becomes scalar exponential times the vector -> components stay in relative proportion



# Review: Eigen Analysis

$$A v_i = \lambda_i v_i$$

$$(A - \lambda_i I) v_i = 0$$

Only has solutions when the determinant of  $A - \lambda I$  is zero.

$$|A - \lambda_i I| = 0$$

For a 2x2 matrix:

$$A = \begin{pmatrix} a & b \\ c & d \end{pmatrix}$$

$$A - \lambda I = \begin{pmatrix} a - \lambda & b \\ c & d - \lambda \end{pmatrix}$$

$$|A - \lambda I| = \begin{vmatrix} a - \lambda & b \\ c & d - \lambda \end{vmatrix}$$

$$= \lambda^2 - (a + d)\lambda + ad - bc$$

Solve via quadratic formula



# Control via Pole Assignment

Given: State-Space System, Desired pole locations (or natural frequency and damping ratio)

Task: Find gains to place poles in desired locations

$$\begin{pmatrix} \Delta \dot{\phi} \\ \Delta \dot{p} \end{pmatrix} = \begin{pmatrix} 0 & 1 \\ \frac{-k_2}{I_x} & \frac{-k_1}{I_x} \end{pmatrix} \begin{pmatrix} \Delta \phi \\ \Delta p \end{pmatrix}$$

$$I_x = 7 \times 10^{-5} \text{ kg m}^2$$

Desired:  $\zeta = 0.7$   
 $\omega_n = 5 \text{ rad/s}$

$$|A - \lambda I| = 0$$

$$\begin{vmatrix} -\lambda & 1 \\ -k_2/I_x & -k_1/I_x - \lambda \end{vmatrix} = \lambda^2 + \frac{k_1}{I_x} \lambda + \frac{k_2}{I_x} = 0$$

$$\lambda = -\zeta \omega_n \pm i \omega_n \sqrt{1 - \zeta^2}$$

$$\lambda = -3.5 \pm 3.57 i$$

Quadratic Formula

$$\lambda = -\frac{k_1}{2 I_x} \pm \sqrt{\frac{k_1^2}{4 I_x^2} - \frac{k_2}{I_x}}$$

$$k_1 = 3.5 \times 2 I_x = 4.9 \times 10^{-4}$$

$$k_2 = -I_x \left( (3.57 i)^2 - \frac{k_1^2}{4 I_x^2} \right) = 0.00175$$





Next Time:  
State space analysis of entire  
quadrotor linear system

