

PH-567: Non-Linear Dynamics

The Duffing Oscillator: From Simple Harmonics to Complex Chaos

Soumik Sahoo Onkar Dagade Vasudev Dubey

Outline

Simple spring-mass system

2 Introducing non-linearity

3 Duffing Oscillator

4 Simulations

5 Analogous analogy

6 Wait a minute...



References

Dynamics of a Duffing Oscililator System

Jongoh (Andy) Jeong Professor Anita Raja Ph235 Physics Simulations 29 April 2019

ve.

The chaotic behavior in motion is not unusual to observe in practice in common porlinear The hoo Survey of Regular and Chaotic Phenomena

in the Forced Duffing Oscillator

YOSHISUKE UEDA

Department of Electrical Engineering. Kyoto University. Kyoto 606, Japan

(Received 11 March 1991)

Abstract - The periodically forced Duffing oscillator

$$\ddot{x} + k\dot{x} + x^3 = B\cos t$$

exhibits a wide variety of interesting phenomena which are fundamental to the behavior of nonlinear dynamical systems, such as regular and chaotic motions, coexisting attractors, regular and fractal basin boundaries, and local and global bifurcations. Analog and digital simulation experiments have provided a survey of the most significant types of behavior; these experiments are essential to any complete understanding, but the experiments alone are not sufficient, and careful interpretation in terms of the geometric theory of dynamical systems is required. The results of the author's survey, begun over 25 yr ago, are here brought together to give a reasonably complete view of the behavior of this important and prototypical dynamical system.

Analogue Electrical Circuit for Simulation of the Duffing-Holmes Equation

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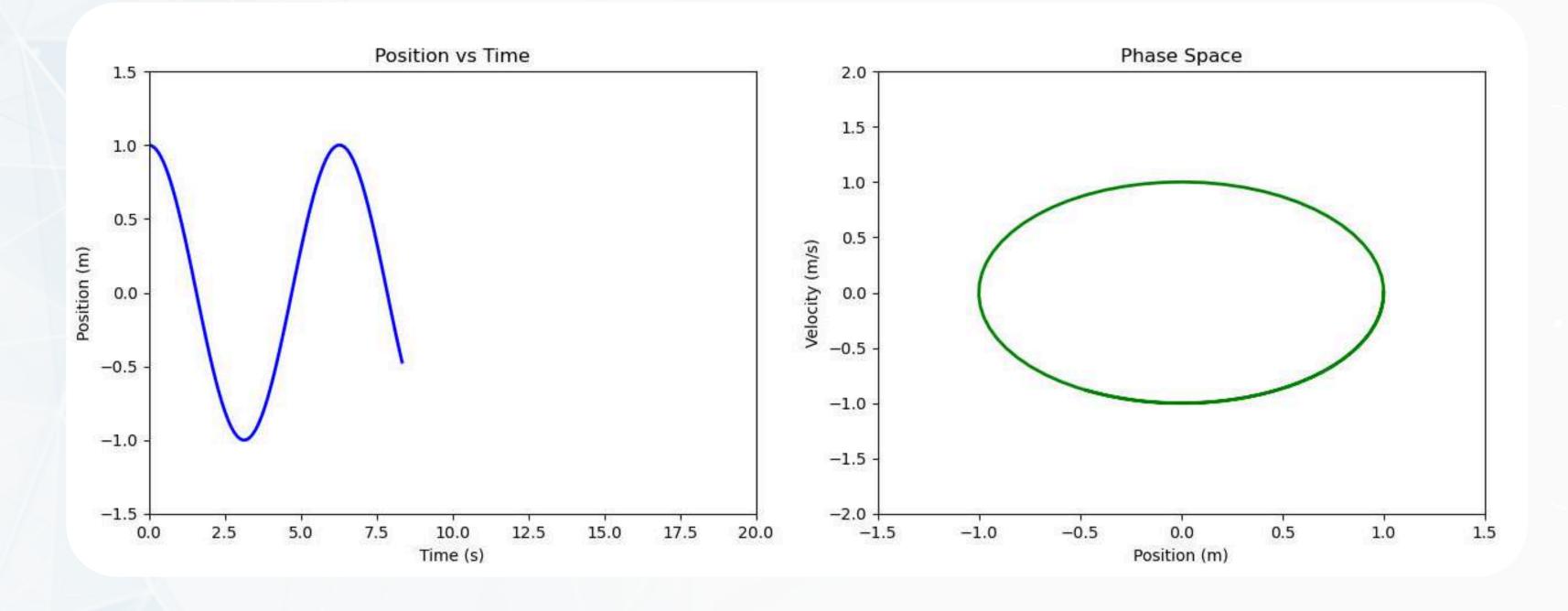
³Ørsted●DTU Department, 348 Technical University of Denmark DK-2800, Lyngby, Denmark el@oersted.dtu.dk

ceived: 15.10.2007 Revised: 31.01.2008 Published online: 02.06.2008

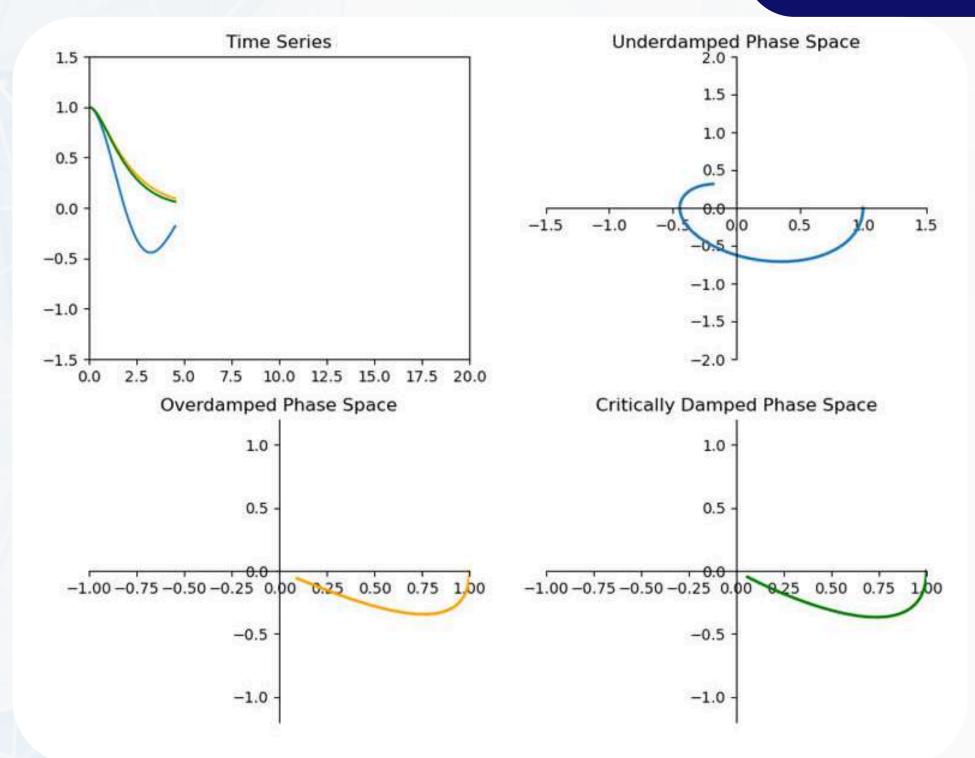
stract. We describe an extremely simple second order analogue electrical circuit for ulating the two-well Duffing-Holmes mathematical oscillator. Numerical results and logue electrical simulations are illustrated with the snapshots of chaotic waveforms, h the phase portraits (the Lissajous figures) and with the stroboscopic maps (the nearé sections).

ywords: nonlinear dynamics, chaos, electrical circuits.

Spring-mass system



Damping + Spring-mass system



$$m\frac{d^2x}{dt^2} + b\frac{dx}{dt} + k_1x = 0$$

$$rac{d^2x}{dt^2} + 2\zetarac{dx}{dt} + \omega_0^2x = 0$$

ullet Underdamping $(\zeta < 1)$

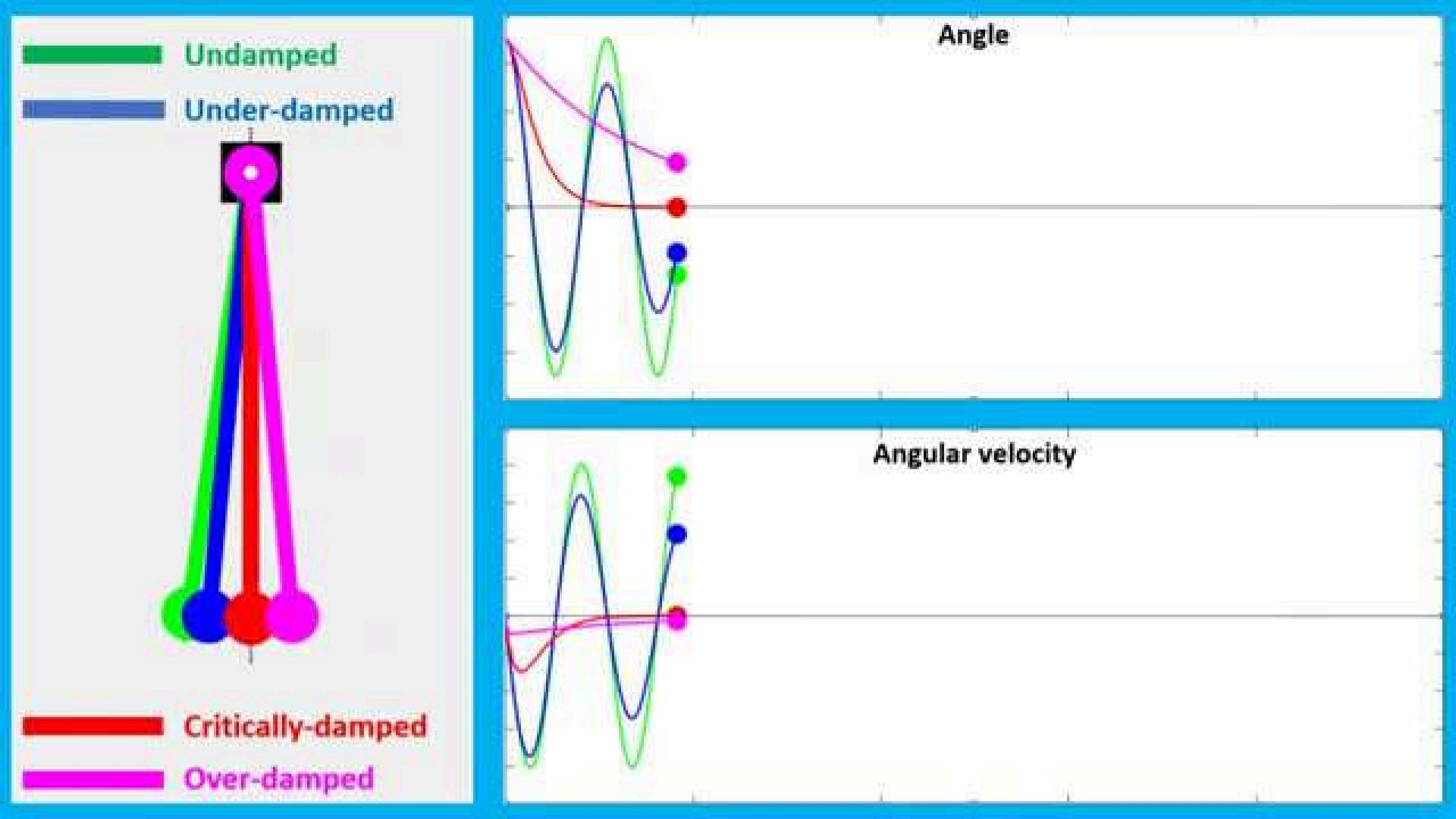
$$x(t) = Ae^{-\zeta\omega_0 t}(cos(\omega t + \phi))$$

• Critically-damping

$$x(t) = (at + b)e^{-\zeta\omega_0 t}$$

Overdamping

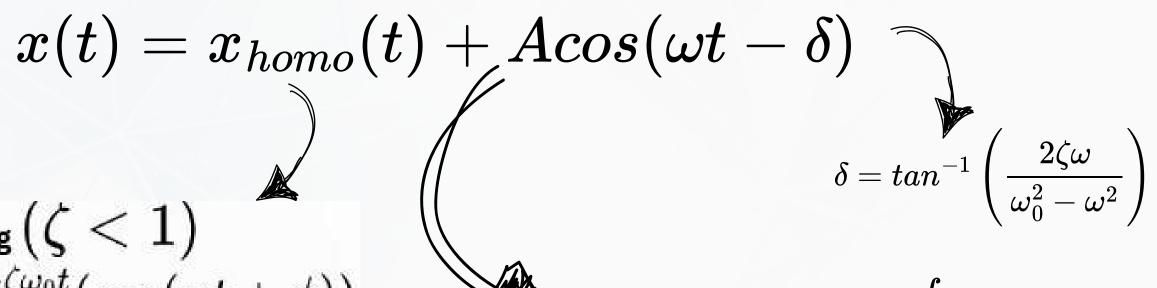
$$x(t) = Ae^{r_1t} + Be^{r_2t}$$



Force + Damping + Spring-Mass system

$$rac{d^2x}{dt^2} + 2\zetarac{dx}{dt} + \omega_0^2x = f(t) = fcos(\omega t)$$

$$x(t) = x_{homo}(t) + Acos(\omega t - \delta)$$



• Underdamping
$$(\zeta < 1)$$
 $x(t) = Ae^{-\zeta\omega_0 t}(cos(\omega t + \phi))$

Critically-damping

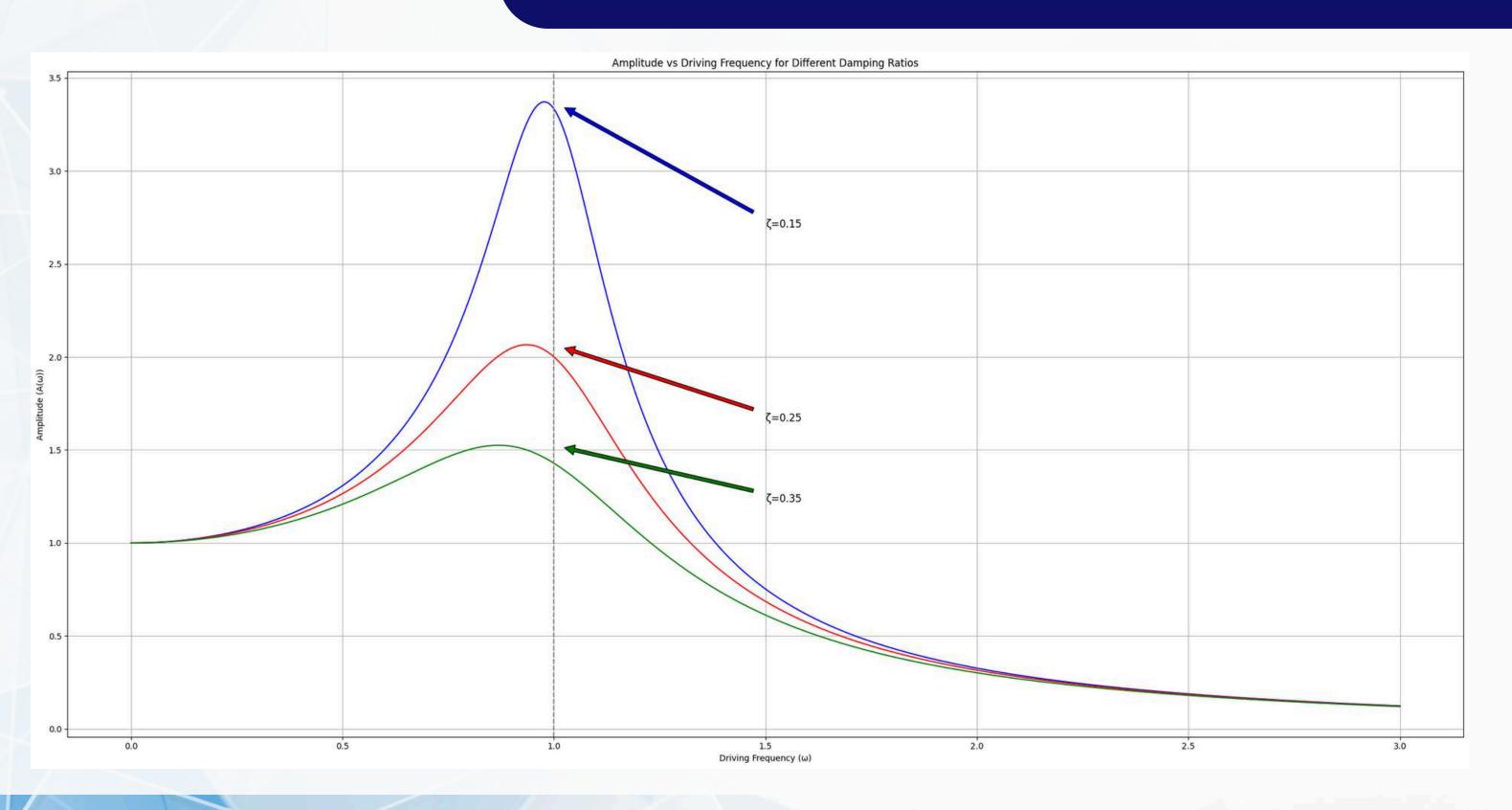
$$x(t) = (at + b)e^{-\zeta\omega_0 t}$$

Overdamping

$$x(t) = Ae^{r_1t} + Be^{r_2t}$$

$$A=rac{f}{\sqrt{(\omega_0^2-\omega^2)^2+(2\zeta\omega_0\omega)^2}}$$

Force + Damping + Spring-Mass system

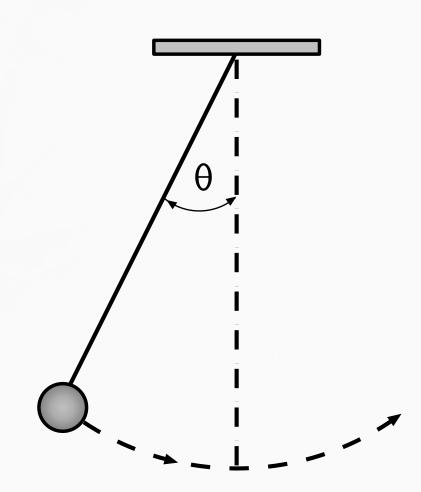


DUFFING OSCILLATOR

$$rac{d^2 heta}{dt^2}+crac{d heta}{dt}=-rac{g}{l}sin heta$$

$$sin heta= heta-rac{ heta^3}{3!}+rac{ heta^5}{5!}-\ldots$$

For simple case we assume for small θ and $sin(\theta) \sim \theta$. But if we add a non linear term θ then ...



Non-autonomous differential equation

$$rac{d^2x}{dt^2} + \deltarac{dx}{dt} + lpha x + eta x^3 = \gamma cos(\omega t)$$

- δ = damping coefficient,
- α= linear stiffness coefficient,
- β = nonlinearity in the restoring force,
- Y = amplitude of the periodic driving force,
- ω = angular frequency of the periodic driving force

Interestingly, the special case with no forcing,

$$\dot{x} = y$$
 $\dot{y} = -\alpha x - \beta x^3$

- Does it mean, energy will be conserved?
- can be integrated by quadratures. Differentiating and plugging in gives

$$\ddot{x} = \dot{y} = -\alpha x - \beta x^3 \Rightarrow \ddot{x}\dot{x} + \alpha \dot{x}x + \beta \dot{x}x^3 = 0$$

• But this can be written as

$$rac{d}{dt}igg(rac{1}{2}\dot{x}^2+lpharac{1}{2}x^2+etarac{1}{4}x^4igg)=0$$

$$\rightarrow$$
 let $(\dots) = h$

$$\dot{x}^2=2h-lpha x^2-etarac{1}{2}x^4$$

$$rac{dx}{dt} = \sqrt{2h - lpha x^2 - eta rac{1}{2} x^4} \qquad t$$

$$rac{dx}{dt} = \sqrt{2h-lpha x^2-eta rac{1}{2} x^4} \qquad t = \int dt = \int rac{dx}{\sqrt{2h-lpha x^2-eta rac{1}{2} x^4}}$$

Note that the invariant of motion satisfies

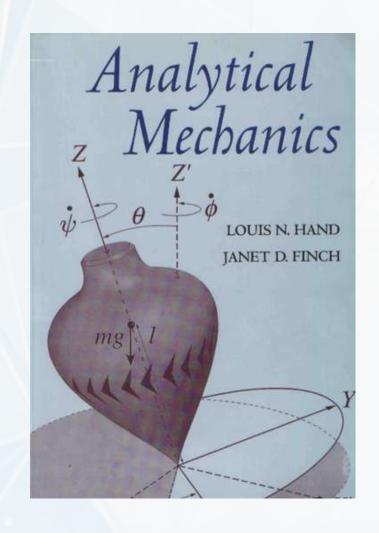
$$\frac{\partial h}{\partial \dot{x}} = \frac{\partial h}{\partial y} = \dot{x}$$

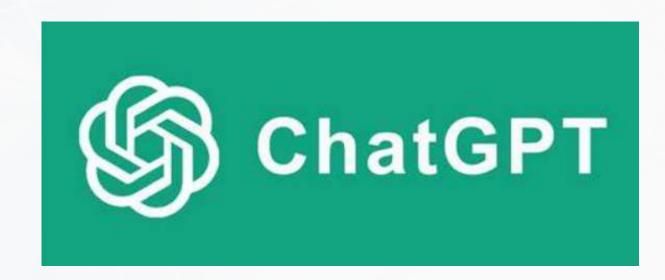
LOOKING SO FAMILIAR:)

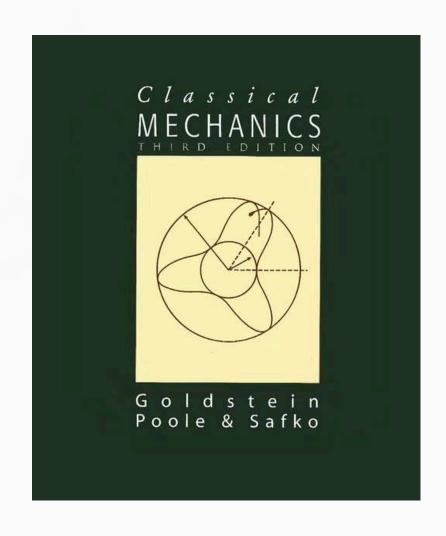
$$rac{\partial h}{\partial x} = lpha x + eta x^3 = -\dot{y}$$

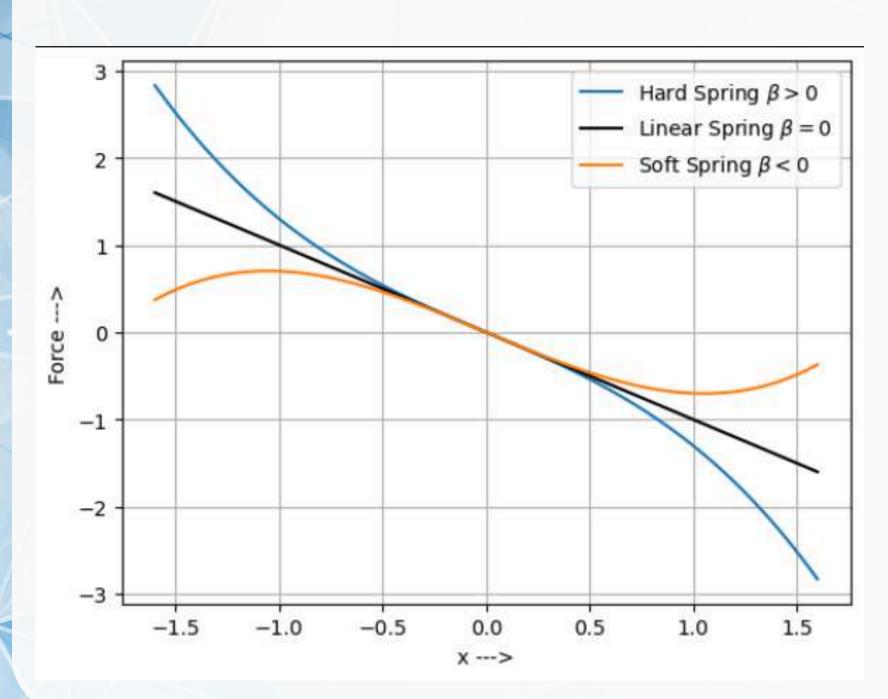
 So the equations of the Duffing oscillator are given by the <u>Hamiltonian system</u> !!!!!!

- Now everyone knows what to do:)
- If not, below are the references:









$$rac{d^2x}{dt^2} + \deltarac{dx}{dt} + lpha x + eta x^3 = \gamma cos(\omega t)$$

- For no damping and zero external force case, and
- For β >0, can be interpreted as a forced oscillator with spring restoring force:

$$F = -\alpha x - \beta x^3$$

Frequency Response

 In the case when there was no anharmonicity (x^3 term), the frequency response was

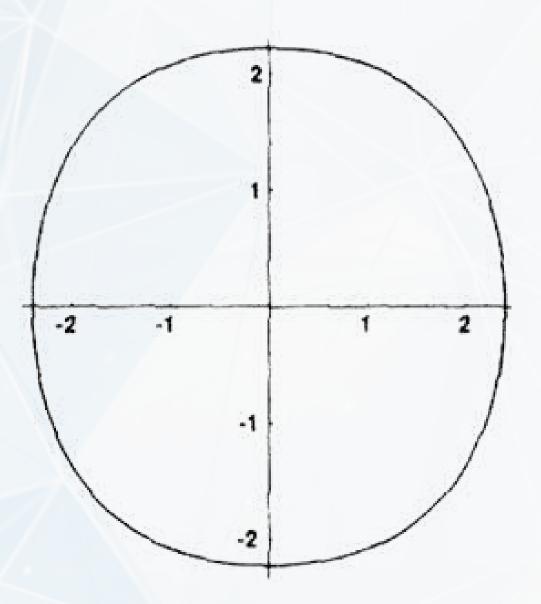
$$A=rac{f}{\sqrt{(\omega_0^2-\omega^2)^2+(2\zeta\omega_0\omega)^2}}$$

After adding non-linearity (taking damping ~ 0),

$$\ddot{x} + 2\zeta\dot{x} + x + \beta x^3 = \gamma cos(\omega t)$$

Harmonic Analysis

• Without non-linear term, we would get a perfect circle.



- Taking β =0.1, Y=1, ω =1 (resonant frequency).
- Introduction of non-linear term resulted in distorted circle.
- We can assume a periodic solution and expand it as a Fourier series. Using symmetry of the equation;

$$x(t) = \sum_{n=1,3,5,...} A_n cos(n\omega t)$$

```
import numpy as np
   from scipy.optimize import fsolve
   # Define parameters
   omega = 1.0 # Angular frequency
   f = 1.0  # External force amplitude
   epsilon = 0.1 # Small nonlinearity parameter
   # Define the equations for A1 and A3
   def equations(vars):
       A1, A3 = vars
       eq1 = (1 - \text{omega}^{**2}) * A1 + (3/4) * epsilon * A1**3 - f
       eq2 = (1 - 9 * omega**2) * A3 + <math>(1/4) * epsilon * A1**3
       return [eq1, eq2]
   # Initial guess for A1 and A3
   initial_guess = [1.78, 0.1]
   # Solve the system of equations
   solution = fsolve(equations, initial_guess)
   A1, A3 = solution
   # Output the solution
   print(f"A1 = {A1}")
   print(f"A3 = {A3}")
 ✓ 0.0s
A1 = 2.371262202993375
A3 = 0.04166666666666666664
```

Harmonic Analysis

By numerically analysis, we found that

$$A_1 \approx 2.37126$$

 $A_3 pprox 0.04166$

So, with good approximation, we can write

$$x(t) = A_1 cos(\omega t) + A_3 cos(3\omega t)$$

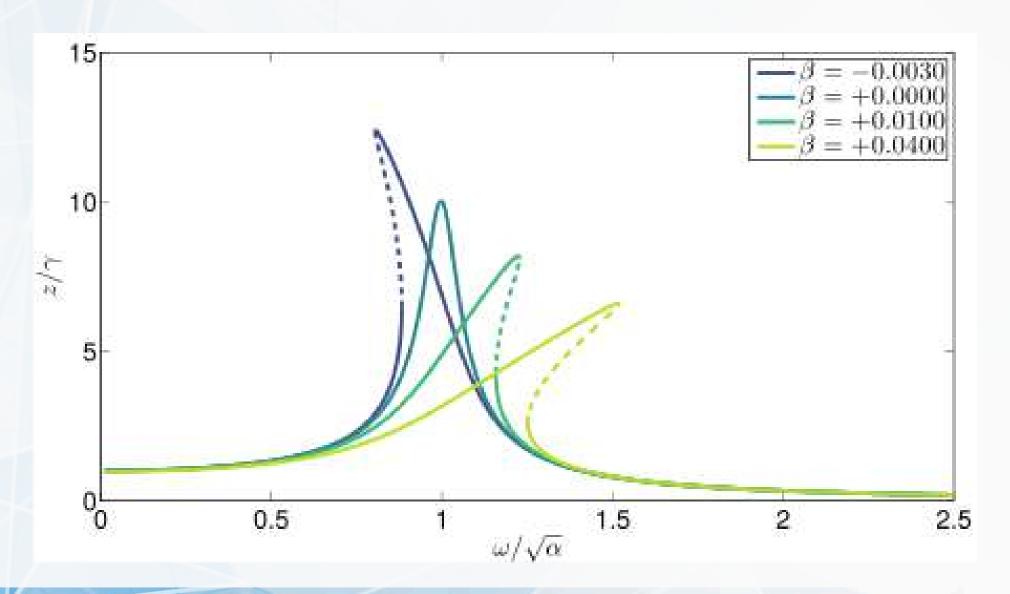
• Putting in the differential equation and equating different coefficients:

$$(1-\omega^2)A_1 + rac{3}{4}eta A_1^3 = \gamma$$

$$(1-9\omega^2)A_3+rac{1}{4}eta A_1^3=0$$

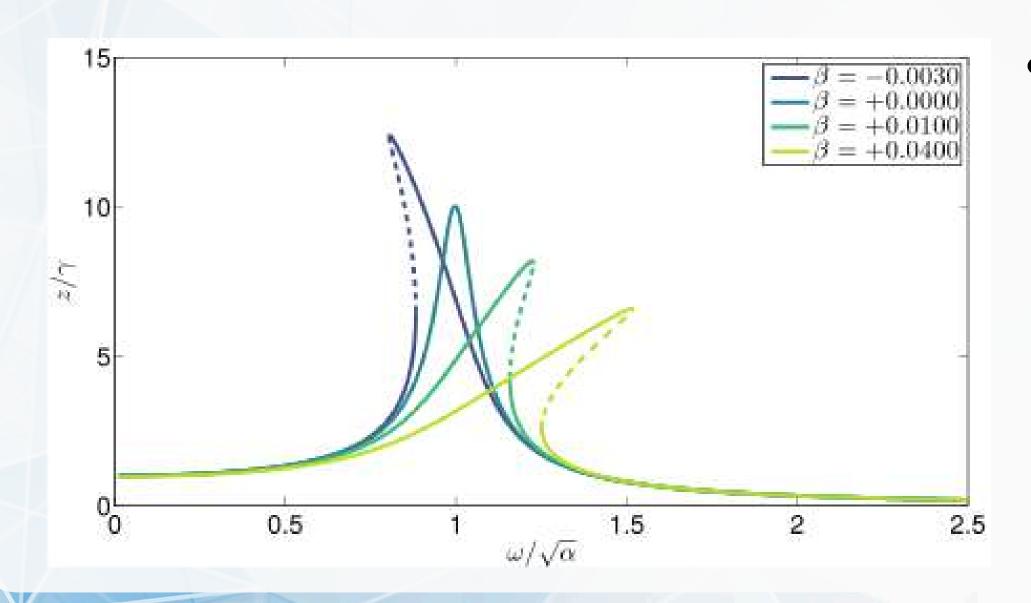
Harmonic Analysis

 For given frequency and force amplitude, we can solve these equations.



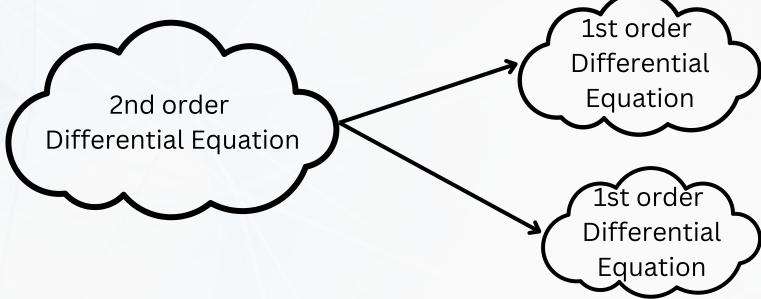
 The cubic term here suggests the possibility of multiple co-existing solutions for a given frequency of external force.

• For hard spring, the effective restoring force increases with increasing displacement, so it's peak will shift towards higher frequency.



• For soft spring, the peak will shift towards lower frequencies.

$$rac{d^2x}{dt^2} + \deltarac{dx}{dt} + lpha x + eta x^3 = \gamma cos(\omega t)$$



$$\dot{x} = y$$
 $\dot{y} = -\alpha x - \beta x^3 - \delta y + \gamma cos(\omega t)$

Simple case ($\gamma = 0$)

The fixed points of the set of coupled differential equation is given by

$$\dot{x}=0, \dot{y}=0$$

$$\dot{m{x}} = m{y} = m{0}$$
 $\dot{m{y}} = -m{x}(lpha + eta m{x}^2) = m{0}$ $X_{fixed} = \{0, \sqrt{-lpha/eta}, -\sqrt{-lpha/eta}\}$ $X_{fixed} = \{0, \sqrt{-lpha/eta}, -\sqrt{-lpha/eta}\}$ $\{(0, 0), (\sqrt{-lpha/eta}, 0), (-\sqrt{-lpha/eta}, 0)\}$

Stability Analysis

- Analysis of the stability of the fixed points can be point by linearizing the equations.
- The equations will be written as:

$$\dot{x}=y$$
 $\dot{y}=-lpha x-eta x^3-\delta y$ $\ddot{y}=-(lpha+3eta x^2)\dot{x}-\delta \dot{y}$

Writing as the matrix form and analysing the stability

$$egin{bmatrix} \dot{x} \ \dot{y} \end{bmatrix} = egin{bmatrix} 0 & 1 \ -lpha - eta x^2 & -\delta \end{bmatrix} egin{bmatrix} x \ y \end{bmatrix}$$

Real eigenvalues λ and μ

$0 < \lambda < \mu$	> /<	Unstable node
$0 < \lambda = \mu$, A diagonalizable		Degenerate unstable node
$0 < \lambda = \mu$, A non-diagonalizable		Unstable node
$\lambda < 0 < \mu$		Saddle
$\lambda = \mu < 0, A$ diagonalizable		Degenerate stable node
$\lambda = \mu < 0, A$ non-diagonalizable		Stable node
$\mu < \lambda < 0$	≯ ≮	Stable node

Complex eigenvalues $(\alpha \pm i\omega, \omega \neq 0)$

$\alpha > 0$	ou O	Unstable spiral
$\alpha = 0$		Center
$\alpha < 0$	ou D	Stable spiral

Stability check at (0,0)

$$egin{array}{c|c} 0-\lambda & 1 \ -lpha & -\delta-\lambda \end{array} = \lambda(\lambda+\delta)+lpha = \lambda^2+\lambda\delta+lpha = 0 \ \lambda_{(0,0)\pm} = rac{1}{2} \left(-\delta\pm\sqrt{\delta^2-4lpha}
ight)$$

 $\mathrm{case}\text{-}1: lpha < 0 \implies \lambda_1 \leq 0, \lambda_2 \geq 0.$ So it's saddle.

case-2 : $\Delta < 0 \implies \Re(\lambda) \le 0$. So it is a focus and stable.

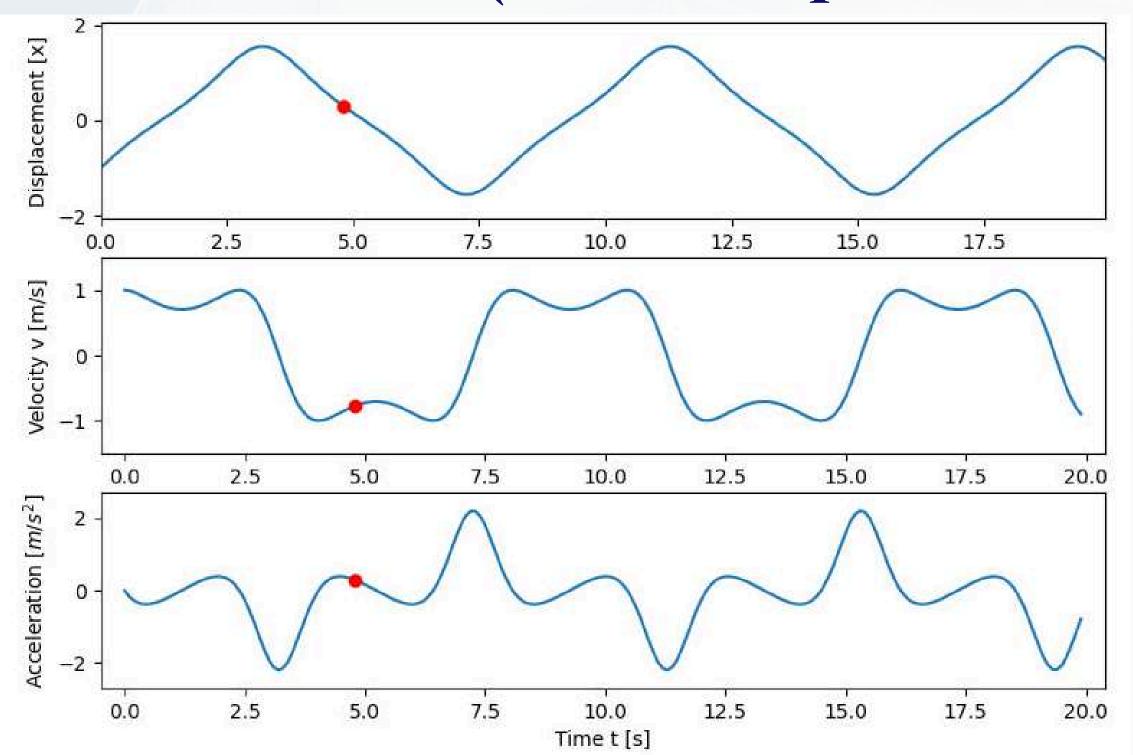
Stability Check at $(\pm \sqrt{-\alpha/\beta}, 0)$

$$egin{bmatrix} \dot{x} \ \dot{y} \end{bmatrix} = egin{bmatrix} 0 & 1 \ -lpha - eta x^2 & -\delta \end{bmatrix} egin{bmatrix} x \ y \end{bmatrix}$$

$$\lambda_{(\pm\sqrt{-lpha/eta},0)} = rac{1}{2} \left(-\delta \pm \sqrt{\delta^2 - 4lpha(1\pm\sqrt{rac{-lpha}{eta}})}
ight)$$

Similarly You can check stablities here for different cases!!

Free Motion (Stable Equilibrium)



Initial conditions:

$$x = -1, v = 1$$

Parameter values:

$$\alpha = -1$$

$$\beta = 1$$

$$\delta = 0$$

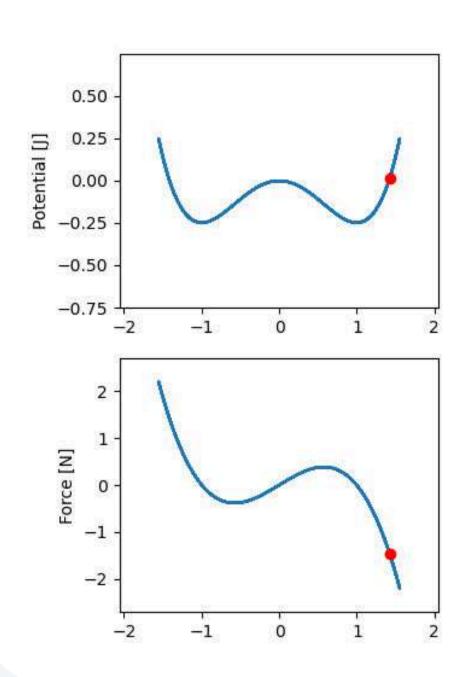
$$\gamma = 0$$

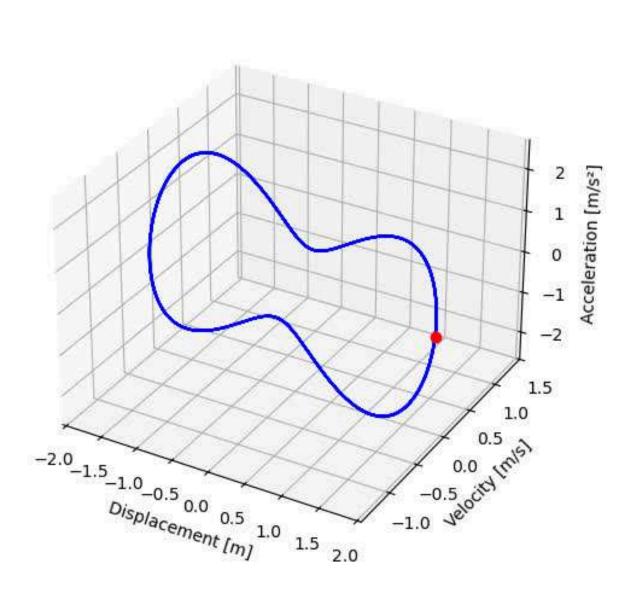
$$\omega = 0$$

$$m = 1$$

Time series

Free Motion (Stable Equilibrium)





Initial conditions:

$$x = -1, v = 1$$

Parameter values:

$$\alpha = -1$$

$$\beta = 1$$

$$\delta = 0$$

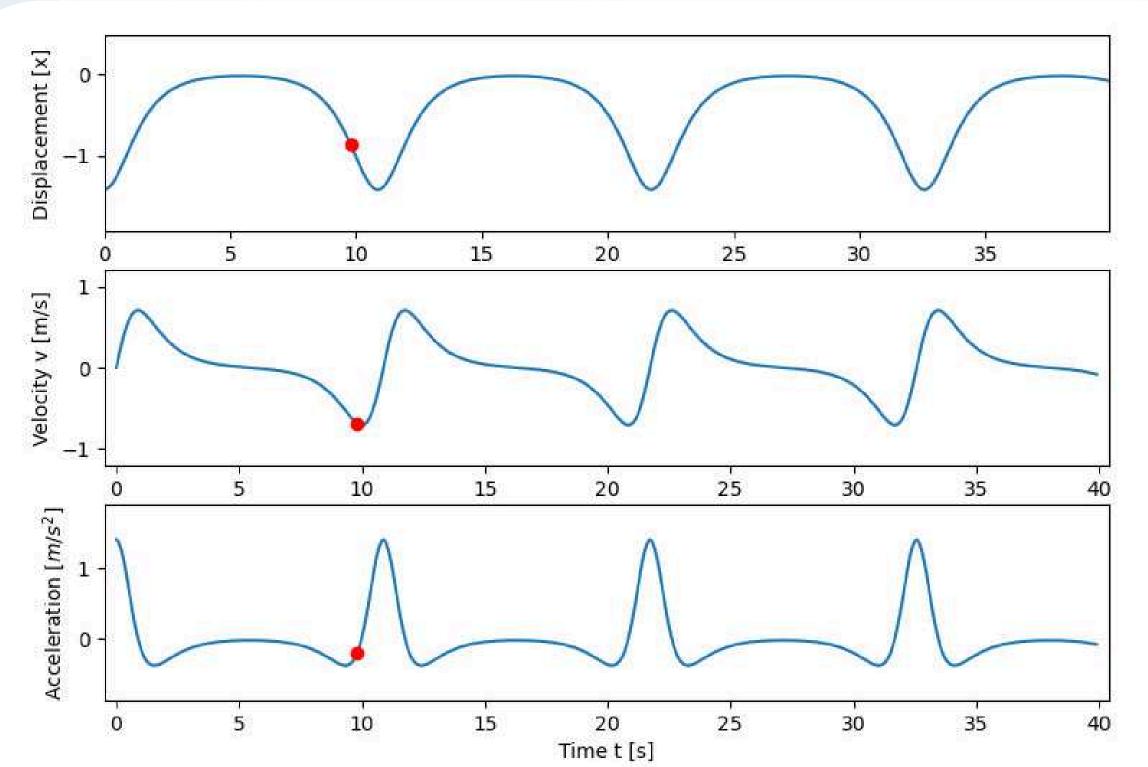
$$\gamma = 0$$

$$\omega = 0$$

$$m = 1$$

PE and Force vs displacement and Phase diagram

Free Motion (Unstable Equilibrium)



Initial conditions:

$$x = -1.414, v = 0$$

Parameter values:

$$\alpha = -1$$

$$\beta = 1$$

$$\delta = 0$$

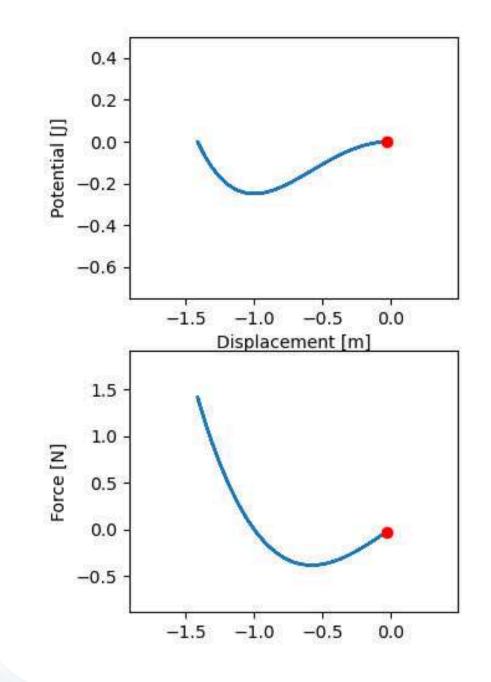
$$\gamma = 0$$

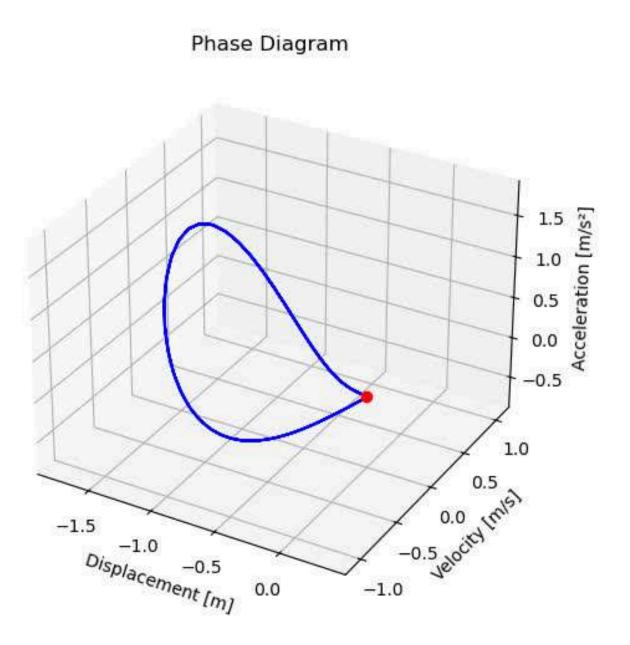
$$\omega = 0$$

$$m = 1$$

Time series

Free Motion (Unstable Equilibrium)





Initial conditions:

$$x = -1.414, v = 0$$

Parameter values:

$$\alpha = -1$$

$$\beta = 1$$

$$\delta = 0$$

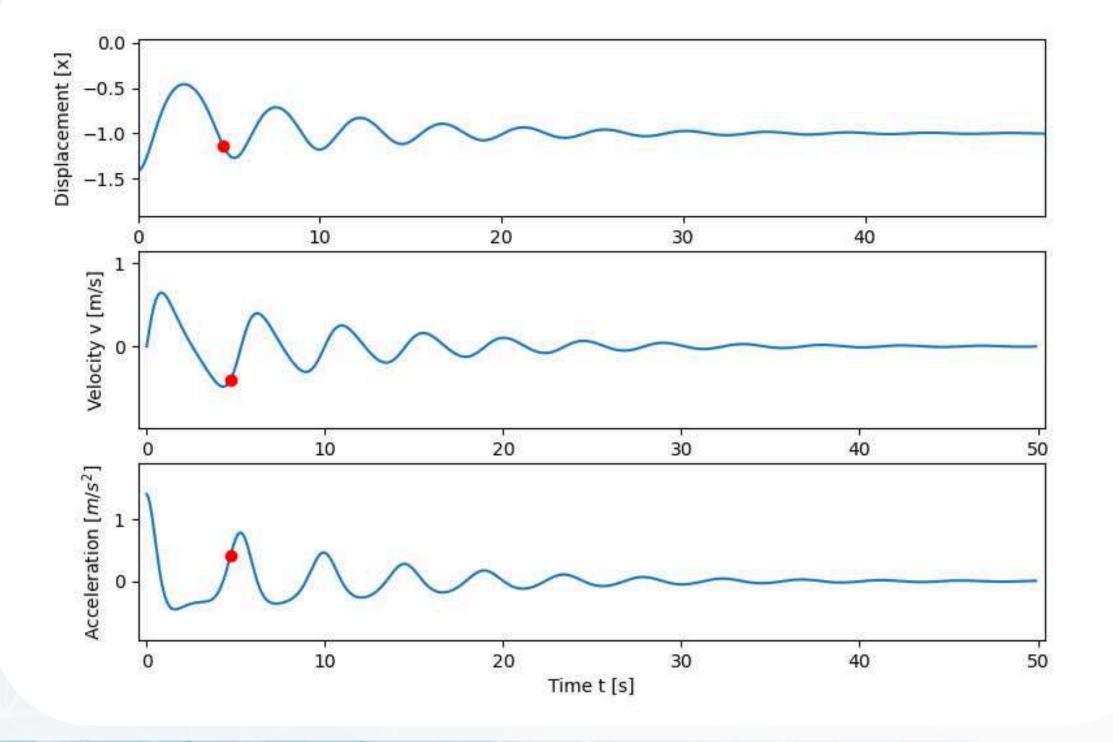
$$\gamma = 0$$

$$\omega = 0$$

$$m = 1$$

PE and Force vs displacement and Phase diagram

Damped Motion



Simulations

Initial conditions:

$$x=-1.414$$
, $v=0$

Parameter values:

$$\alpha = -1$$

$$\beta = 1$$

$$\delta=0.2$$

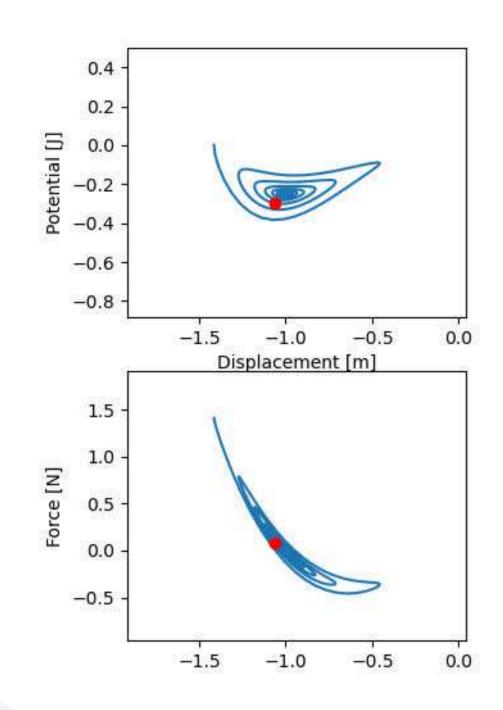
$$\gamma = 0$$

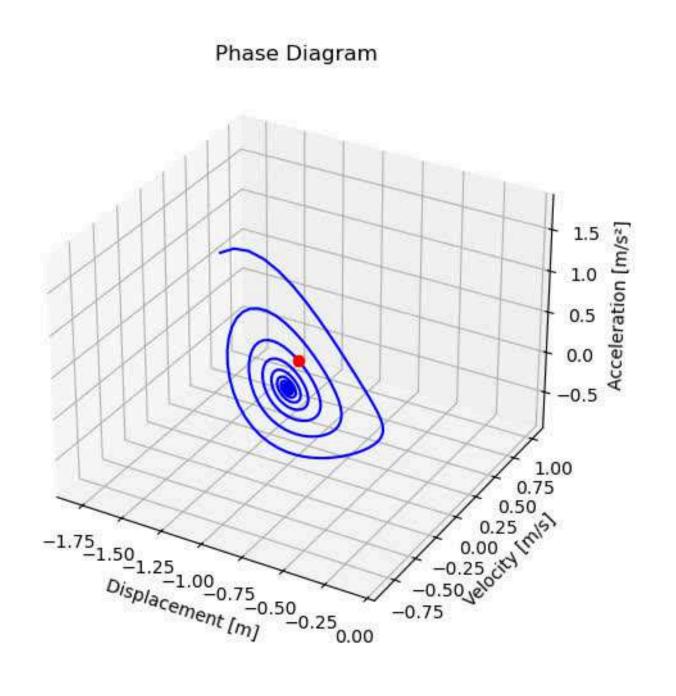
$$\omega = 0$$

$$m = 1$$

Time series

Damped Motion





Initial conditions:

$$x=-1.414$$
, $v=0$

Parameter values:

$$\alpha = -1$$

$$\beta = 1$$

$$\delta=0.2$$

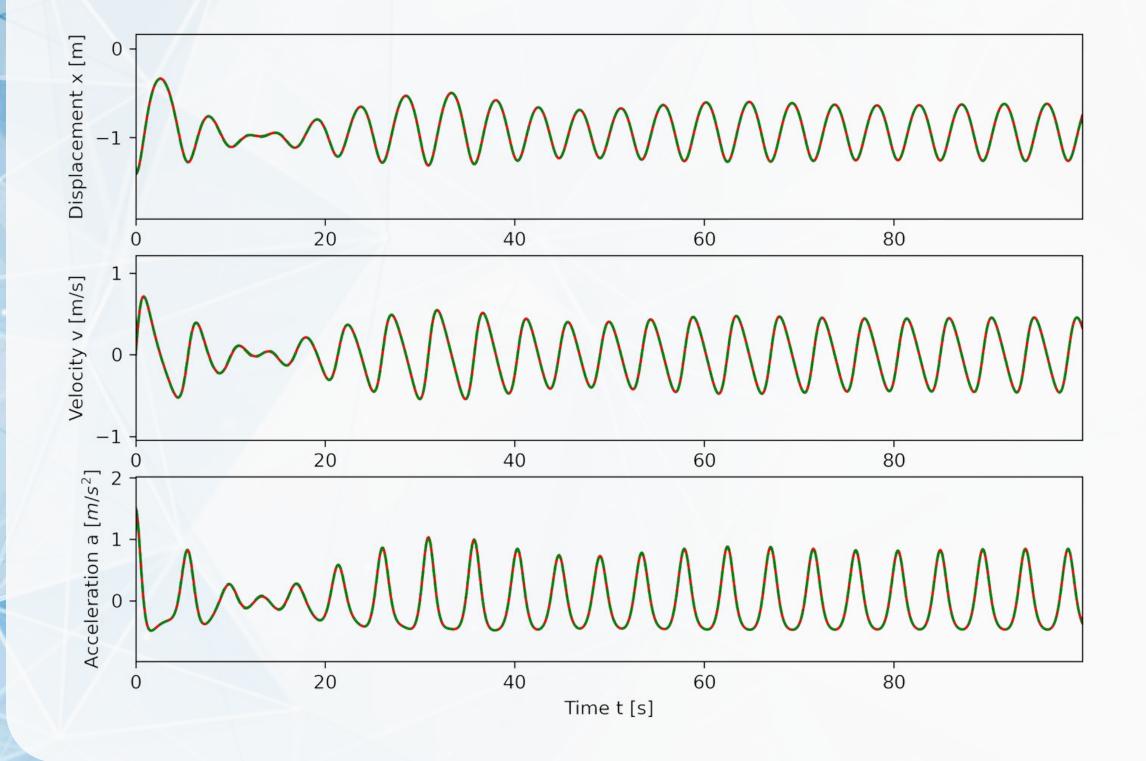
$$\gamma = 0$$

$$\omega = 0$$

$$m = 1$$

PE and Force vs displacement and Phase diagram

Forced Damped Motion



Simulations

Initial conditions:

$$x = -1.414, v = 0$$

$$x = -1.413, v = 0$$

Parameter values:

$$\alpha = -1$$

$$\beta = 1$$

$$\delta = 0.1$$

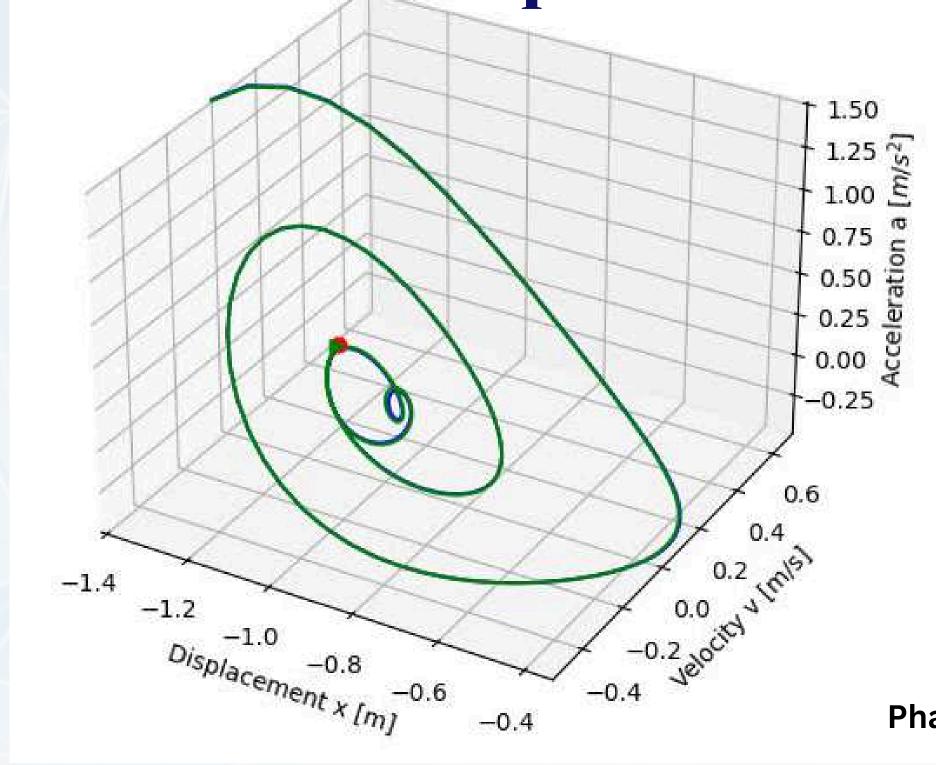
$$\gamma=0.1$$

$$\omega=1.4$$

$$m = 1$$

Time series





Initial conditions:

$$x = -1.414, v = 0$$

$$x = -1.413, v = 0$$

Parameter values:

$$\alpha = -1$$

$$\beta = 1$$

$$\delta = 0.1$$

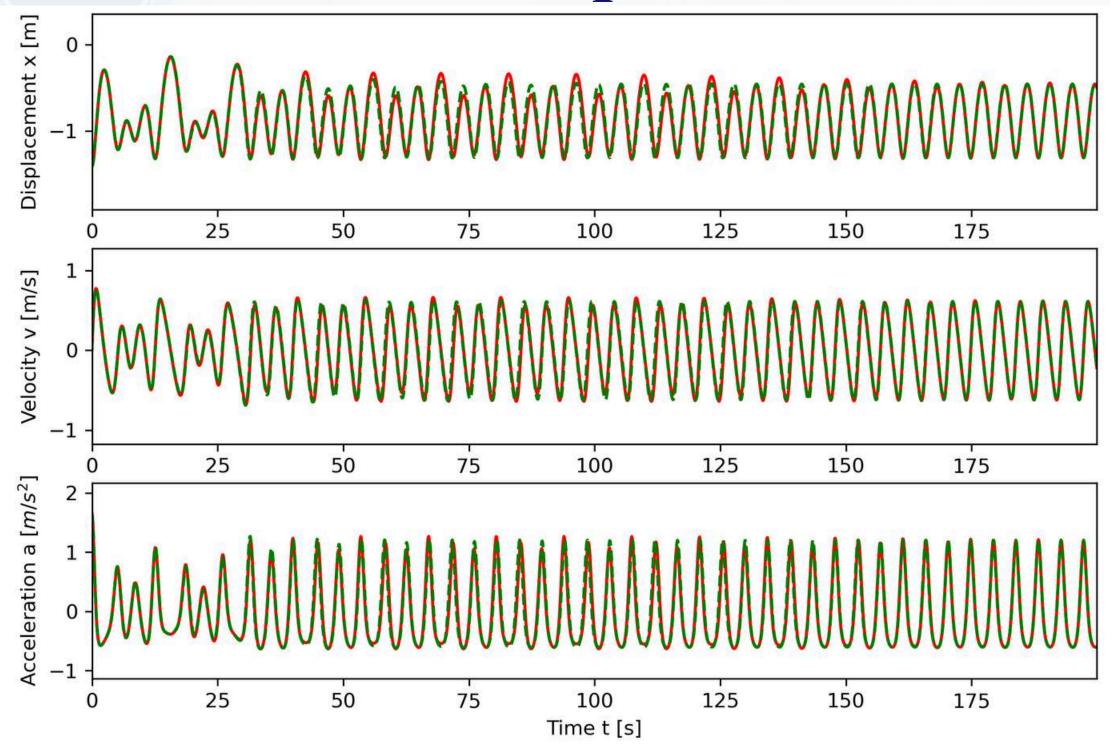
$$\gamma=0.1$$

$$\omega = 1.4$$

$$m = 1$$

Phase diagram

Forced Damped Motion



Initial conditions:

$$x = -1.414, v = 0$$

$$x = -1.413, v = 0$$

$$\alpha = -1$$

$$\beta = 1$$

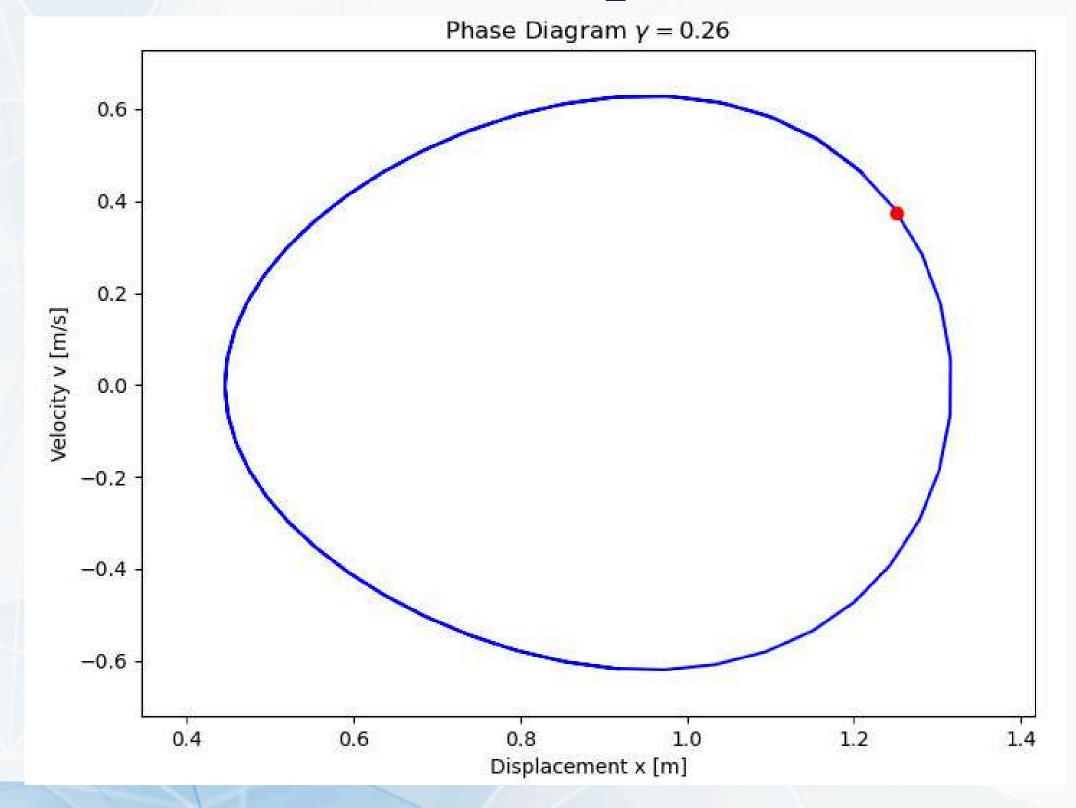
$$\delta = 0.1$$

$$\gamma = 0.26$$

$$\omega=1.4$$

$$m = 1$$

Forced Damped Motion



Simulations

Initial conditions:

$$x = -1.414, v = 0$$

Parameter values:

$$\alpha = -1$$

$$\beta = 1$$

$$\delta=0.1$$

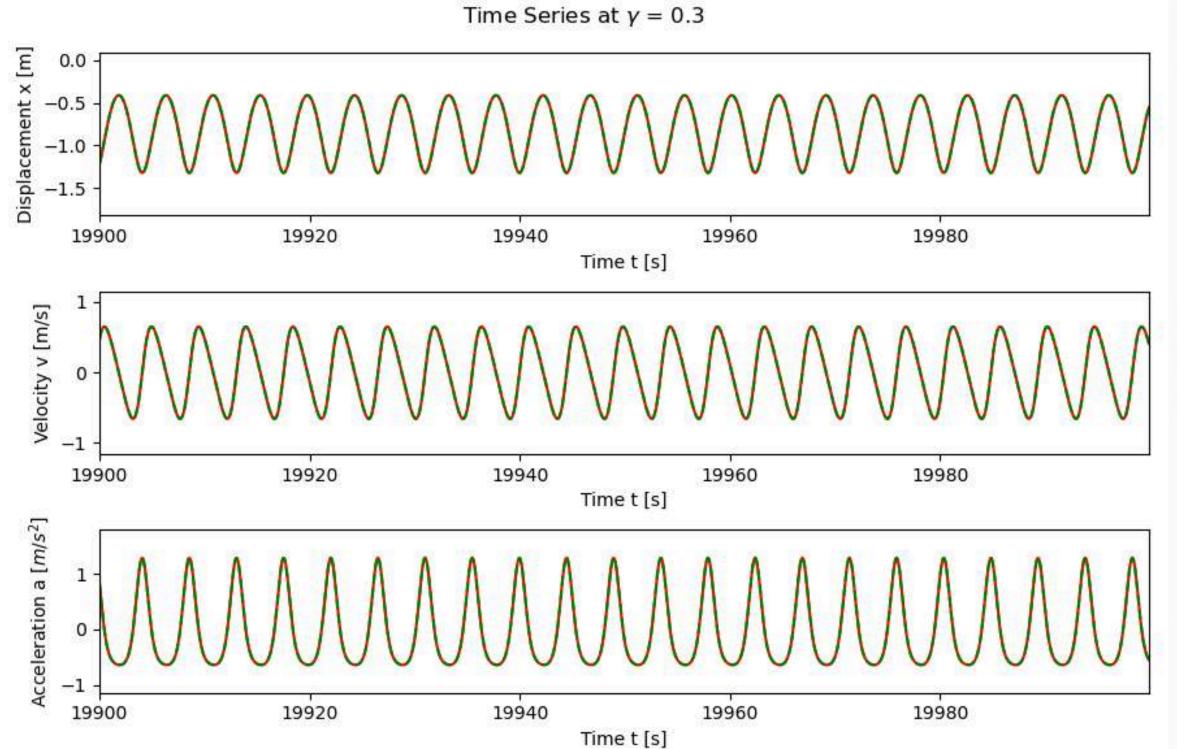
$$\gamma=0.26$$

$$\omega = 1.4$$

$$m = 1$$

Phase diagram

Forced Damped Motion



Initial conditions:

$$x = -1.414, v = 0$$

$$x = -1.413, v = 0$$

$$\alpha = -1$$

$$\beta = 1$$

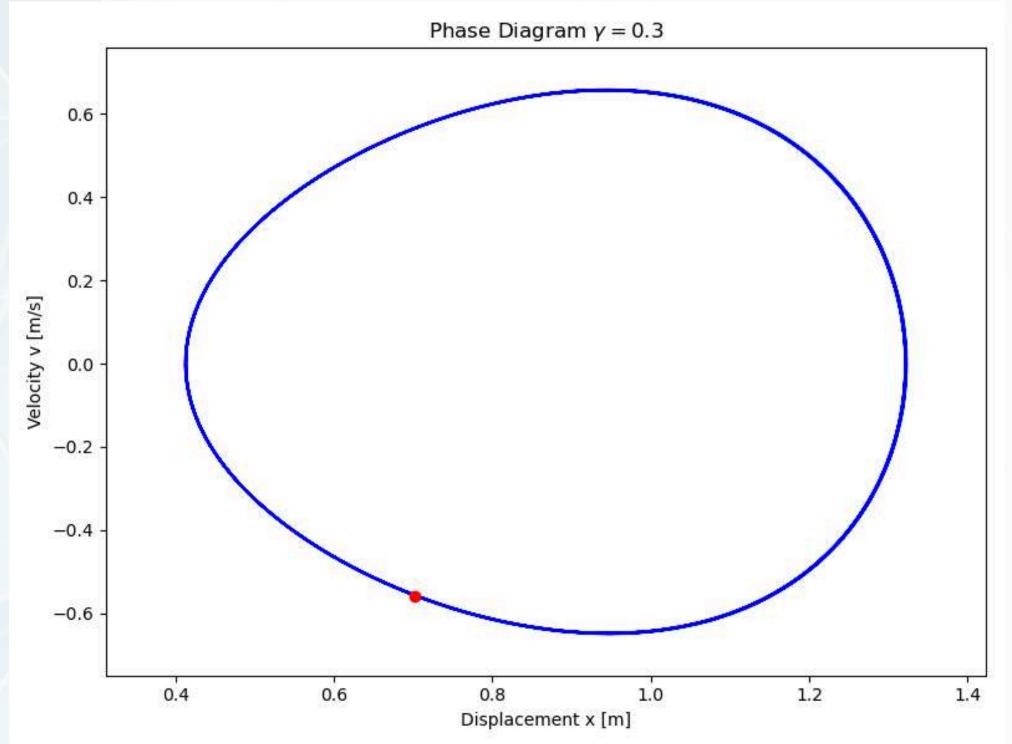
$$\delta = 0.1$$

$$\gamma=0.30$$

$$\omega=1.4$$

$$m = 1$$

Forced Damped Motion



Simulations

Initial conditions:

$$x = -1.414, v = 0$$

$$x = -1.413, v = 0$$

$$\alpha = -1$$

$$\beta = 1$$

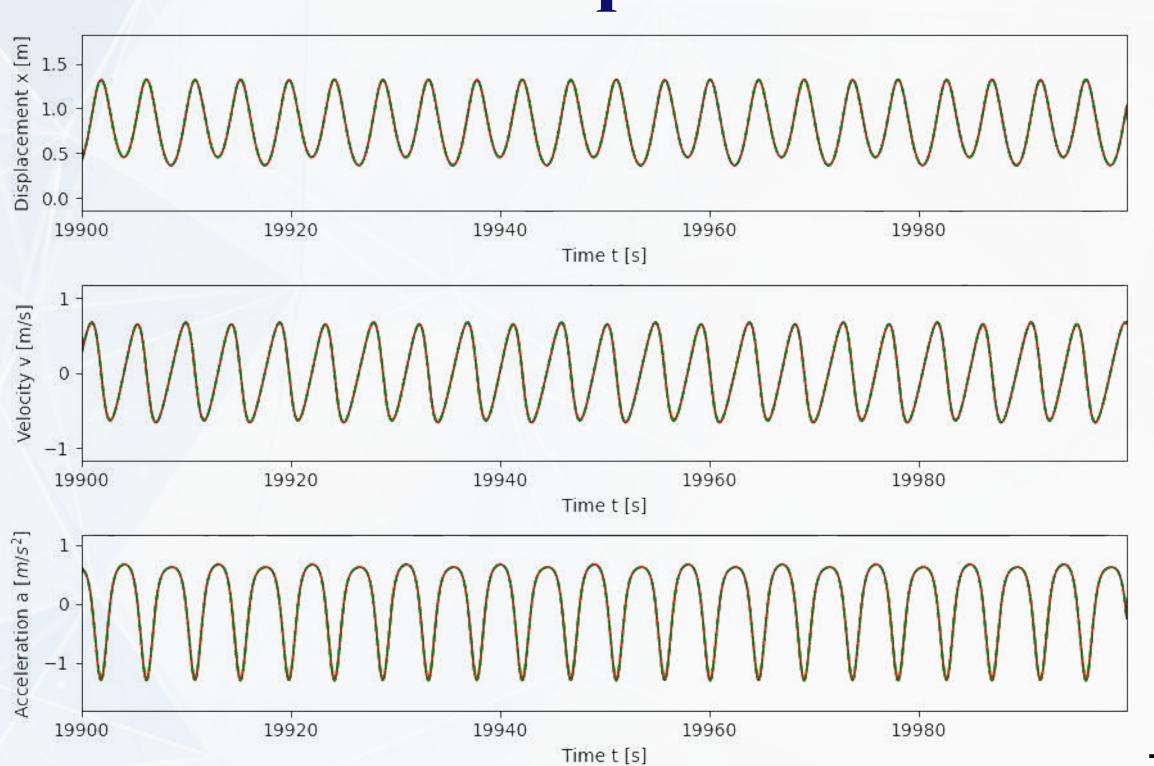
$$\delta=0.1$$

$$\gamma=0.30$$

$$\omega=1.4$$

$$m = 1$$

Forced Damped Motion



Initial conditions:

$$x = -1.414, v = 0$$

$$x = -1.413, v = 0$$

Parameter values:

$$\alpha = -1$$

$$\beta = 1$$

$$\delta = 0.1$$

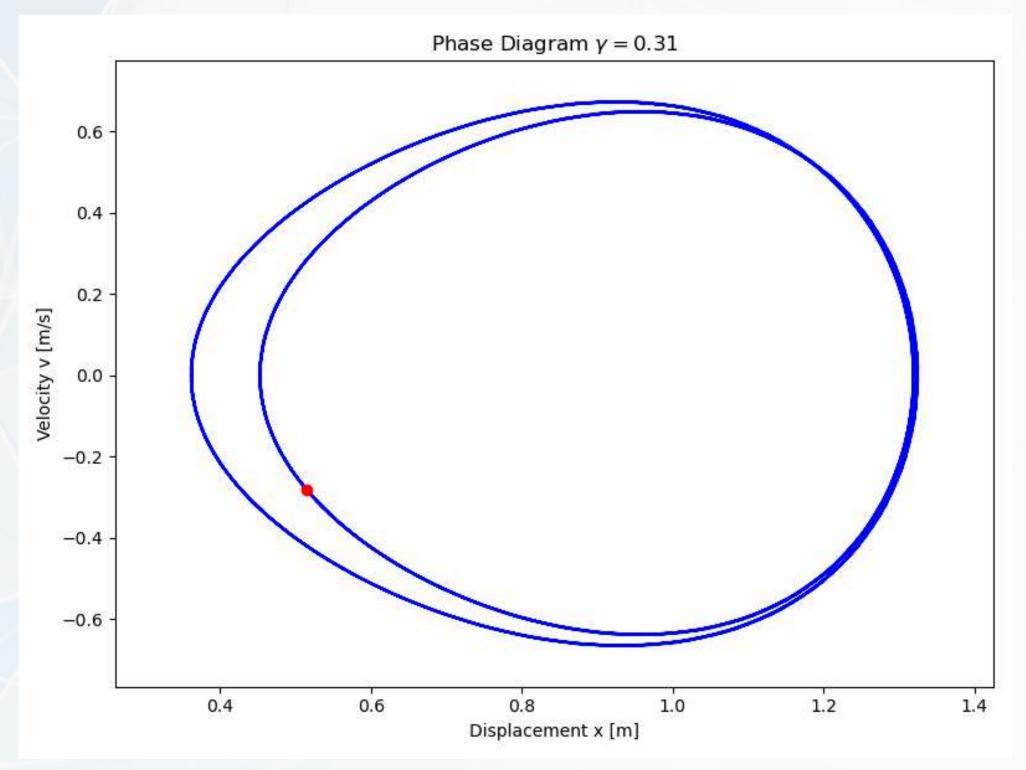
$$\gamma=0.31$$

$$\omega = 1.4$$

$$m = 1$$

Time Series

Forced Damped Motion



Simulations

Initial conditions:

$$x = -1.414, v = 0$$

$$\alpha = -1$$

$$\beta = 1$$

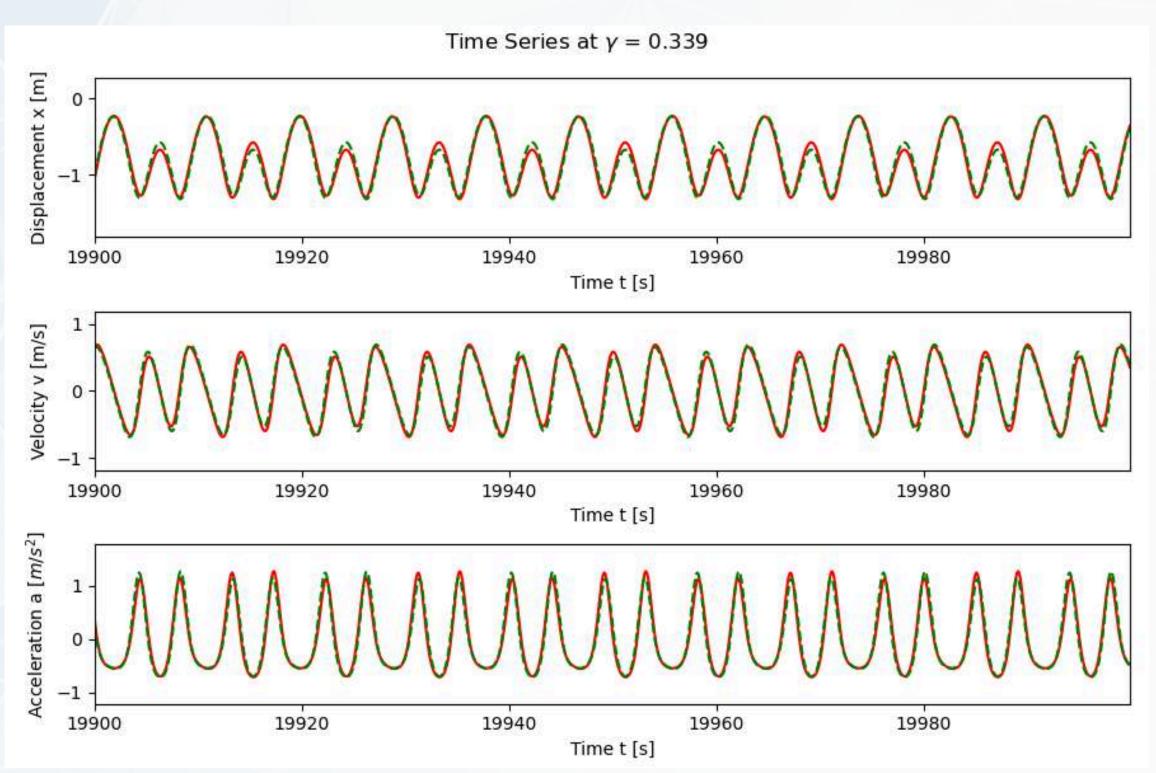
$$\delta=0.1$$

$$\gamma=0.31$$

$$\omega=1.4$$

$$m = 1$$

Forced Damped Motion



Initial conditions:

$$x = -1.414, v = 0$$

$$x = -1.413, v = 0$$

$$\alpha = -1$$

$$\beta = 1$$

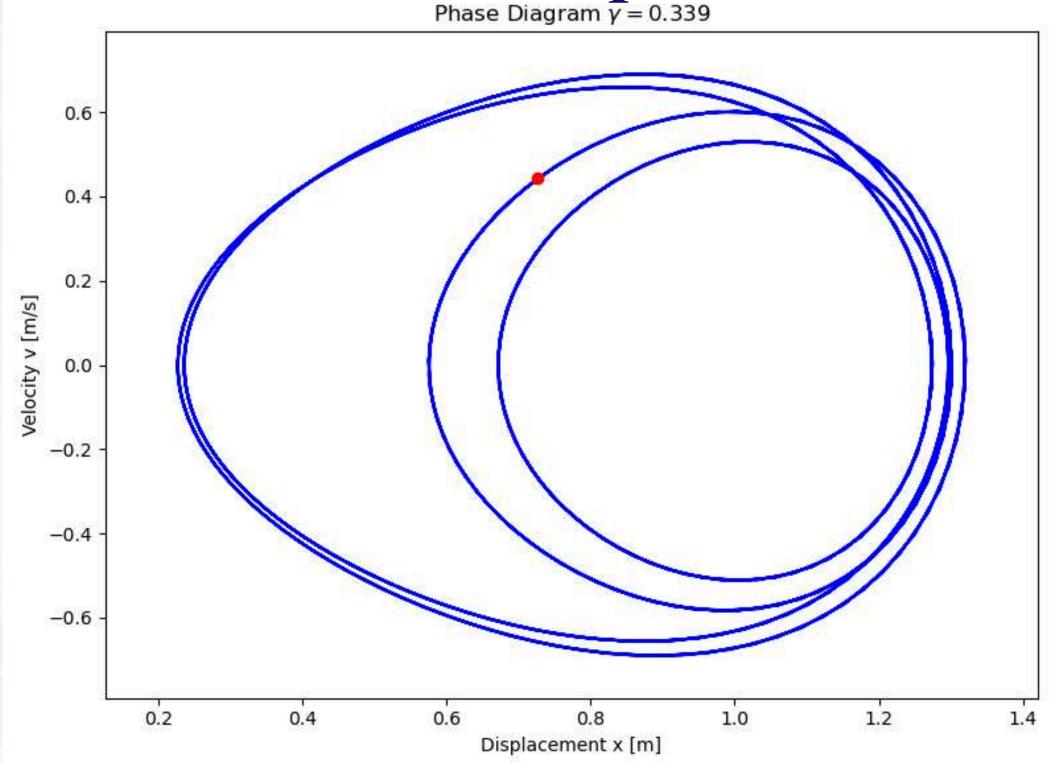
$$\delta=0.1$$

$$\gamma = 0.339$$

$$\omega=1.4$$

$$m = 1$$

Forced Damped Motion Phase Diagram y = 0.339



Simulations

Initial conditions:

$$x = -1.414, v = 0$$

$$\alpha = -1$$

$$\beta = 1$$

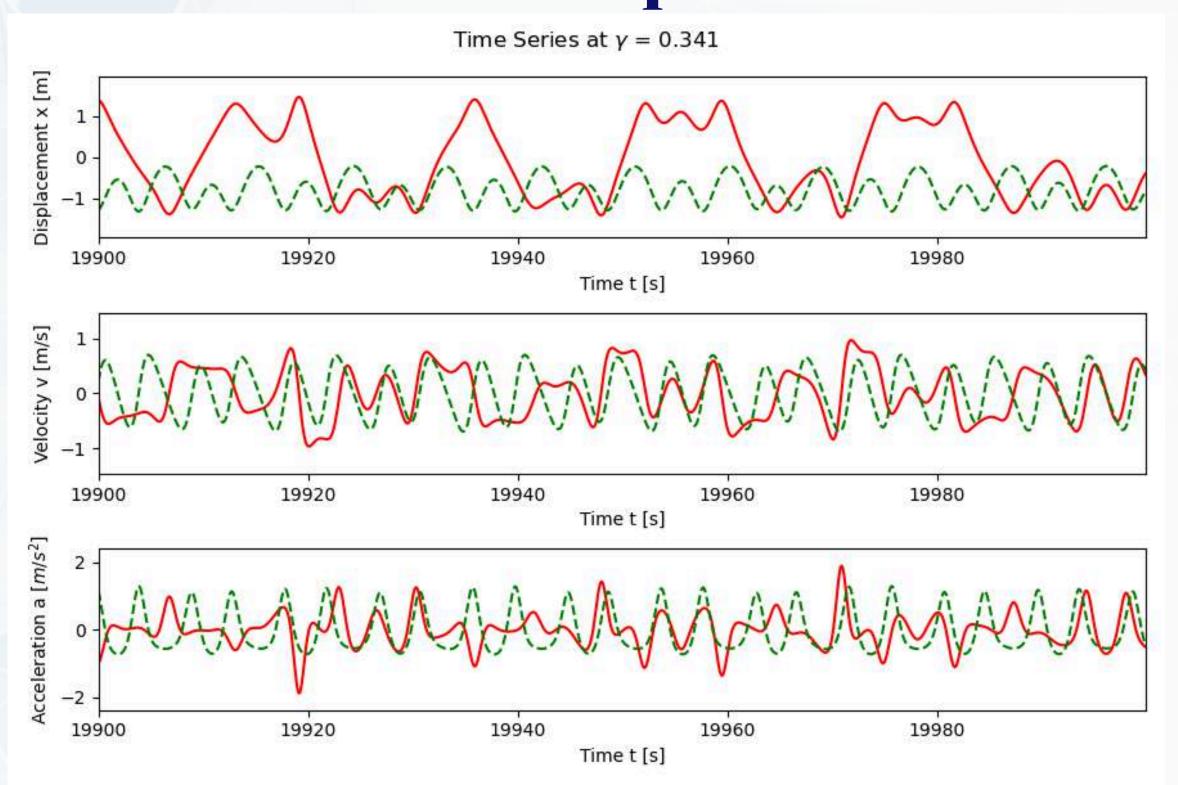
$$\delta=0.1$$

$$\gamma=0.339$$

$$\omega=1.4$$

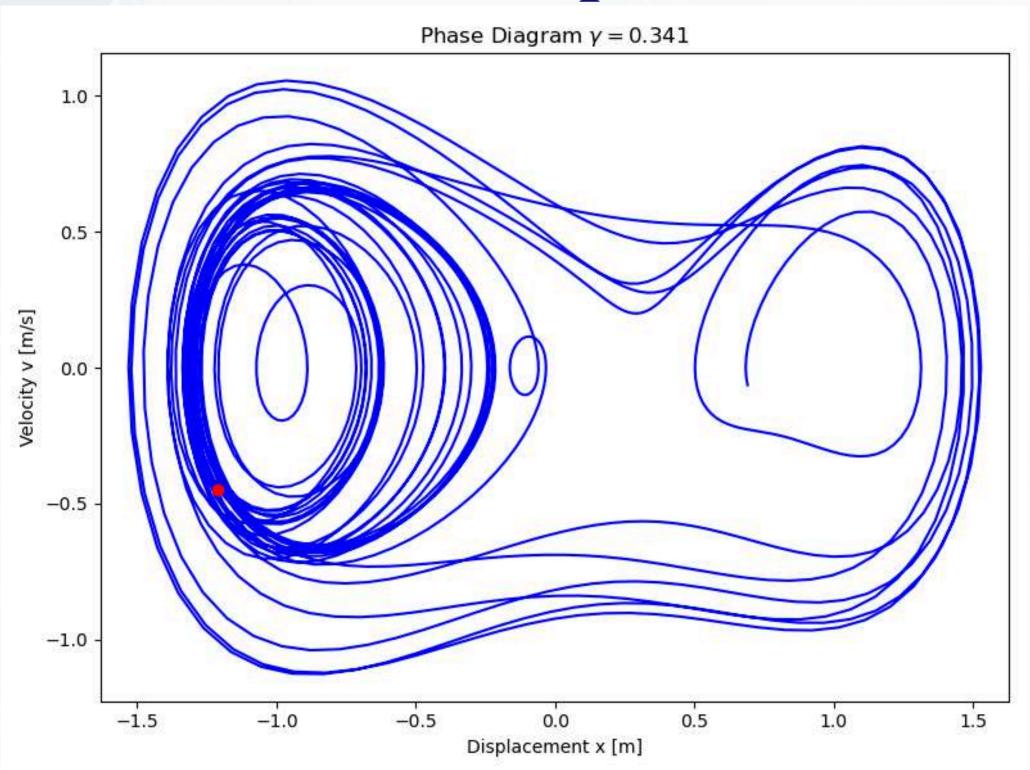
$$m = 1$$

Forced Damped Motion



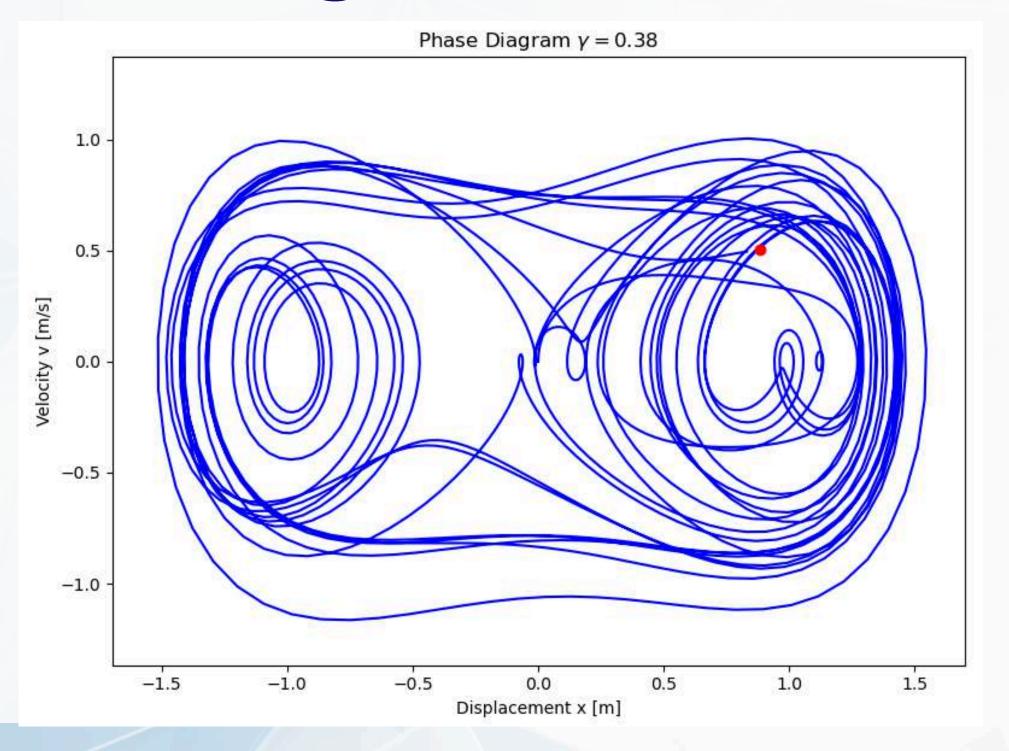
$$egin{aligned} lpha &= -1 \ eta &= 1 \ \delta &= 0.1 \ \gamma &= 0.341 \ \omega &= 1.4 \ m &= 1 \end{aligned}$$

Forced Damped Motion



$$egin{aligned} lpha &= -1 \ eta &= 1 \ \delta &= 0.1 \ \gamma &= 0.341 \ \omega &= 1.4 \ m &= 1 \end{aligned}$$

Period Doubling and ... CHAOS



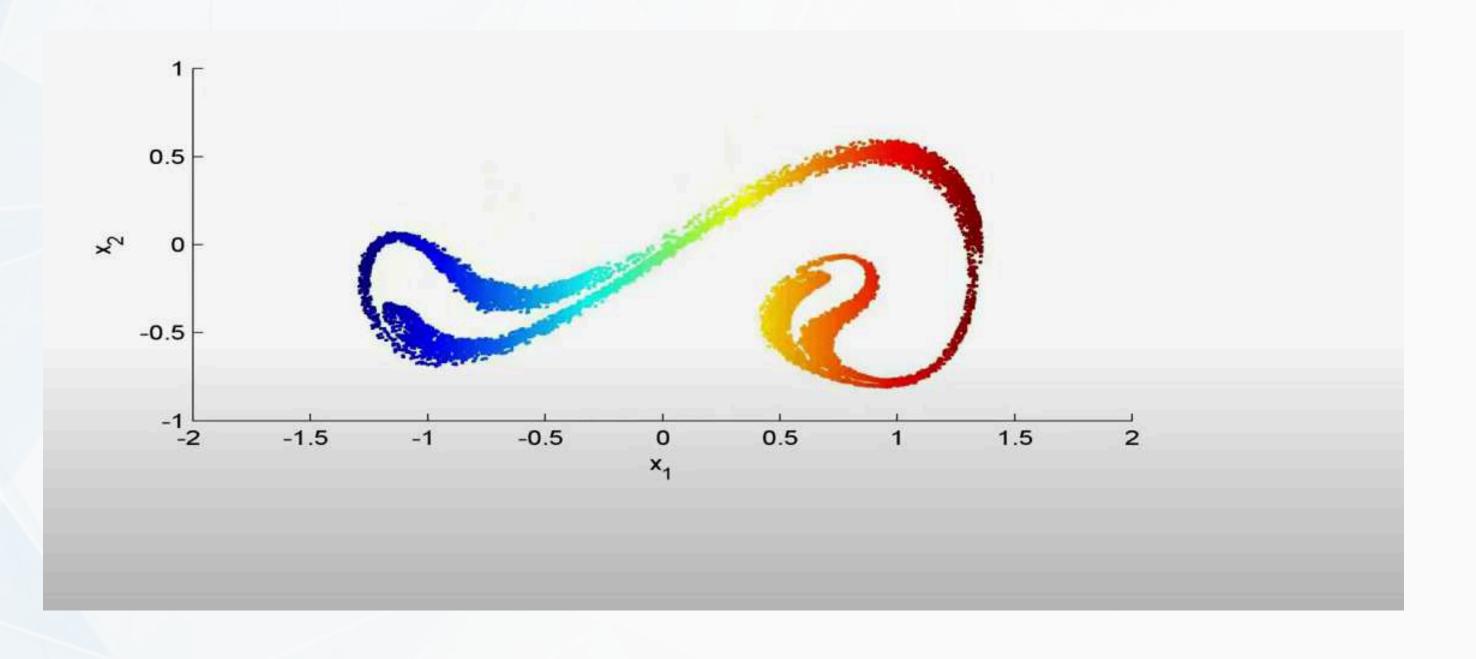
Period Doubling and ... CHAOS

How the Poincaré section will look like?

Let's see (Enjoy the music!)



Period Doubling and ... CHAOS



Analogous Circuit

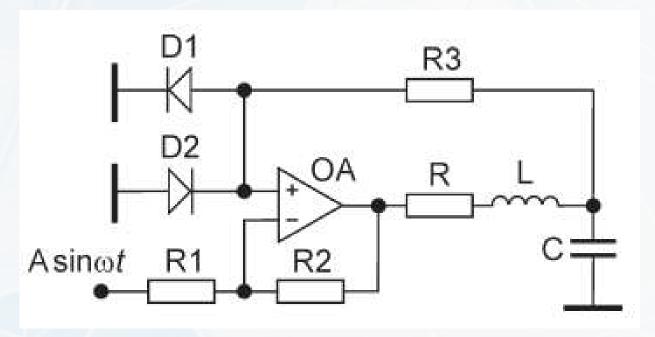
The non-linear equation we mentioned so far in this session has a analogous electrical circuit which show same kind of phenomena.

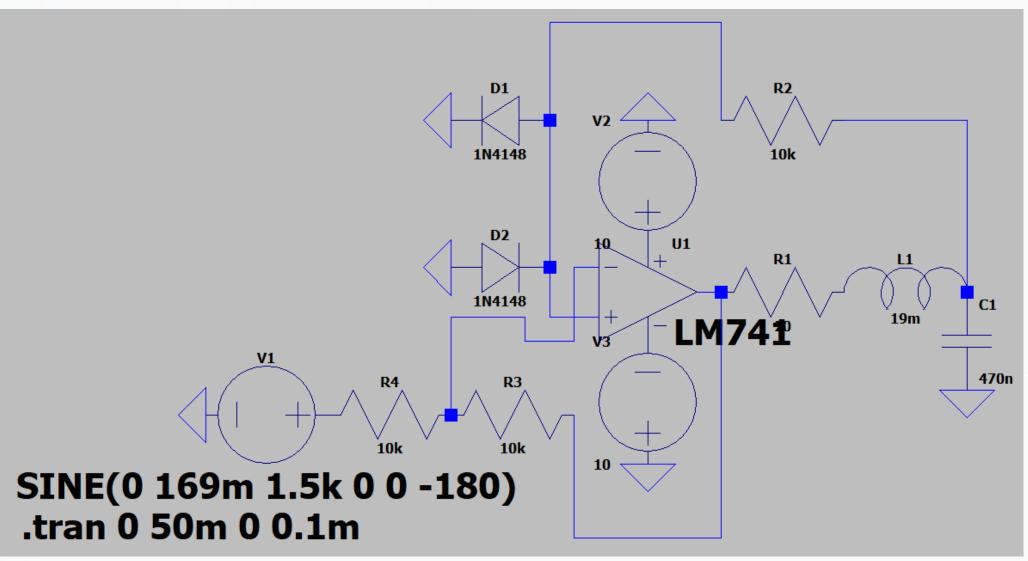
If we put $\alpha = -1$ and $\beta = 1$ then we get a equation Famously known as Duffing-Holmes equation

And we can make a analogous circuit to see the dynamics of the system.

Analogous Circuit

$$rac{d^2x}{dt^2} + \deltarac{dx}{dt} - x + x^3 = \gamma cos(\omega t)$$
 with: $lpha = -1, eta = 1$





Analogous Circuit

$$Crac{dV_C}{dt} = I_L, Lrac{dI_L}{dt} = F_E(V_C) - I_L R + A \sin(\omega t - \pi)$$

 V_C is the voltage across capacitor C, and I_L is the current through inductor L.

Assuming
$$R_3 \gg
ho = \sqrt{rac{L}{C}}$$
, we can ignore the phase π in $A\sin(\omega t - \pi)$.

The function $F_E(V_C)$ is approximated as:

$$F_E(V_C) = egin{cases} -(V_C + kV^*), & V_C < -V^*, \ (k-1)V_C, & -V^* \le V_C \le V^*, \ -(V_C - kV^*), & V_C > V^* \end{cases}$$

Analogous Circuit

 $k = \frac{R_2}{R_1} + 1$ is the gain of the amplifying stage.

 V^* is the voltage drop across an opened diode (for silicon diodes V^*

 $\approx 0.5 \, V \, {
m at} \, 0.1 \, {
m mA}). {
m Choose} \, k = 2 \, {
m by setting} \, R_2 = R_1.$

assume $R_{d0} \gg R_3 \gg R_{d1}$, where R_{d0} and

 R_{d1} are the resistances of the diode in the closed and opened states, respectively.

Introducing dimensionless variables and parameters:

Analogous Circuit

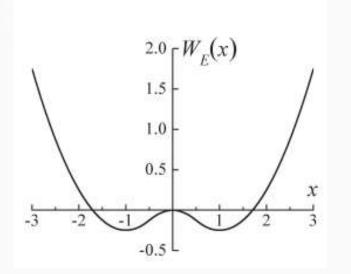
$$x=rac{V_C}{2V^*},\quad y=rac{
ho I_L}{2V^*},\quad rac{t}{\sqrt{LC}} o t,\quad \omega \sqrt{LC} o \omega, \ a=rac{A}{2V^*},\quad b=rac{R}{
ho},\quad
ho=\sqrt{rac{L}{C}},$$

equations convenient for analysis and numerical simulation are obtained:

$$\dot{x}=y, \ \dot{y}=F_E(x)-by+a\sin\omega t$$

Analogous Circuit

$$F_E(x) = egin{cases} -(x+1), & x < -0.5, \ x, & -0.5 \le x \le 0.5, \ -(x-1), & x > 0.5. \end{cases}$$



It mimics like the actual potential but not exact

$$W_E(x) = -\int F_E(x)\, dx = rac{1}{2} egin{cases} (x+1)^2 - 0.5, & x < -0.5, \ -x^2, & -0.5 \le x \le 0.5, \ (x-1)^2 - 0.5, & x > 0.5. \end{cases}$$

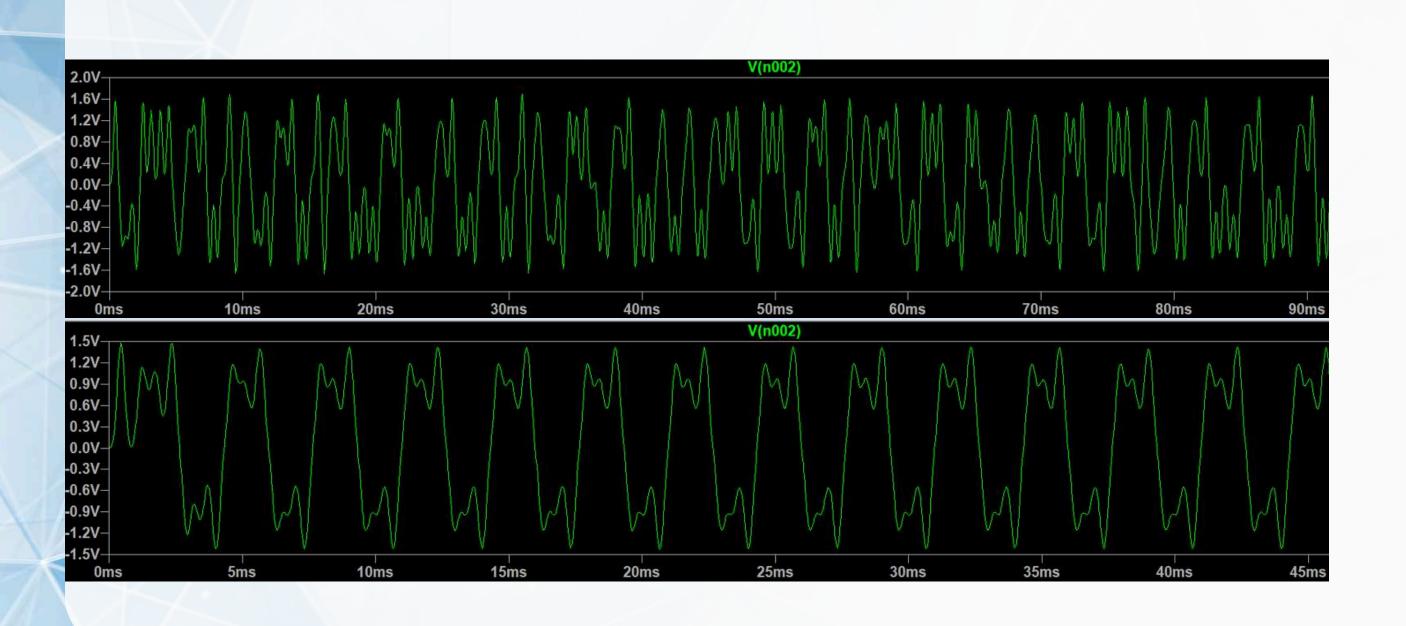
Analogous Circuit



Period-1

Period-2

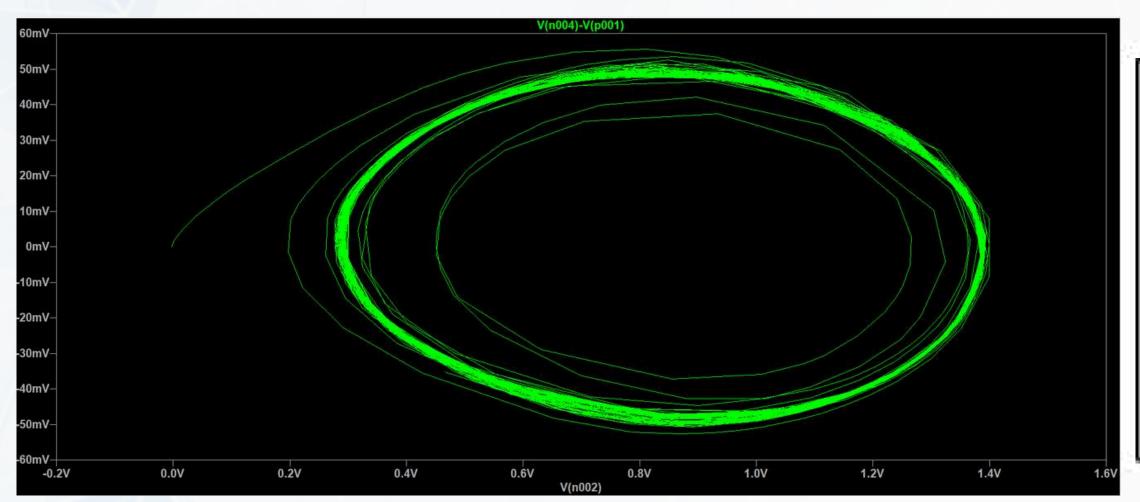
Analogous Circuit

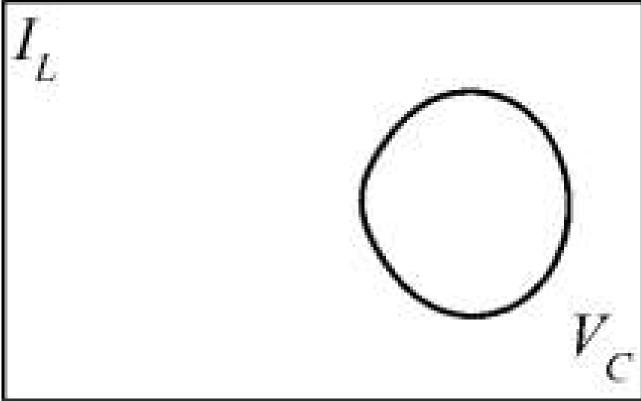


Chaos

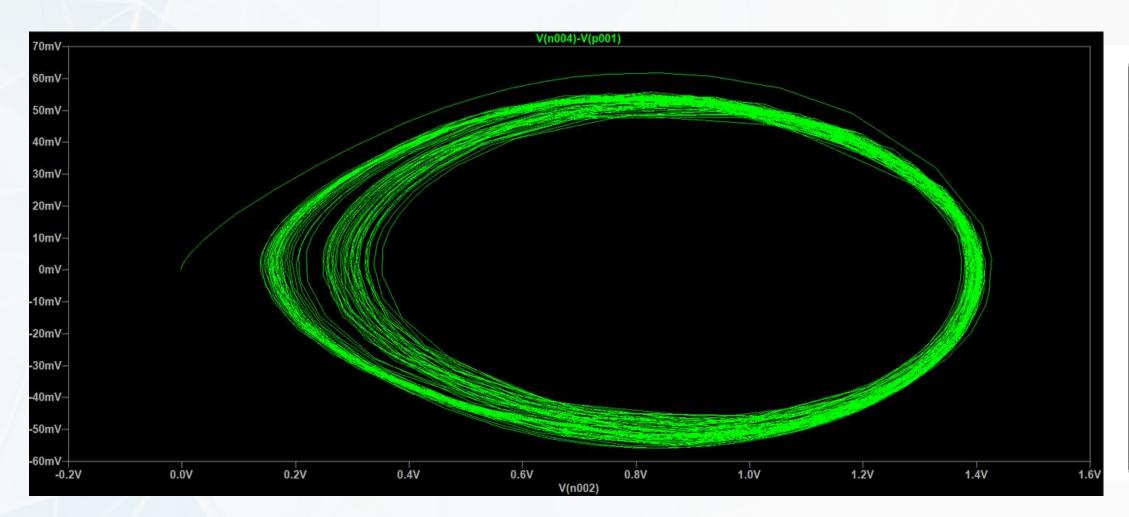
Period-5

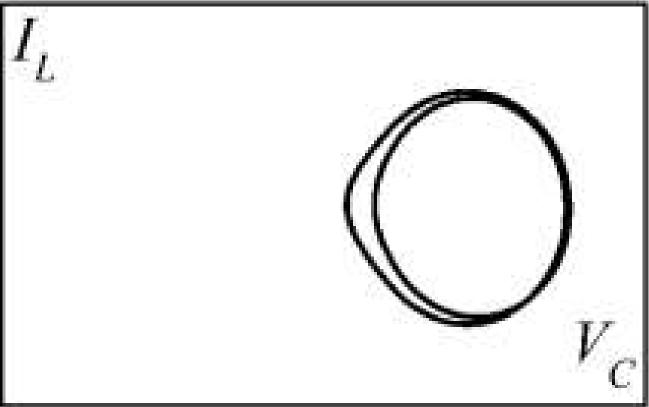
iL vs Vc (Period-1)at Vc=120m



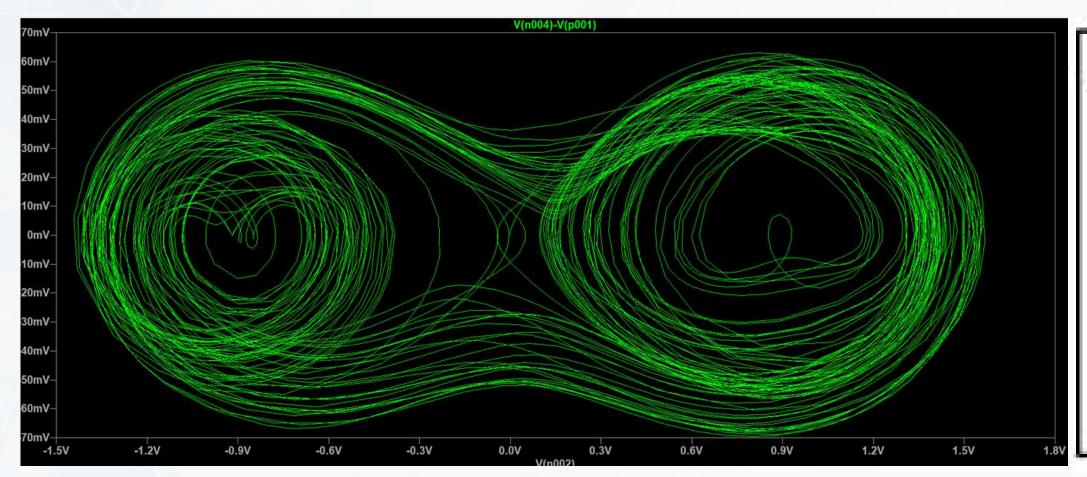


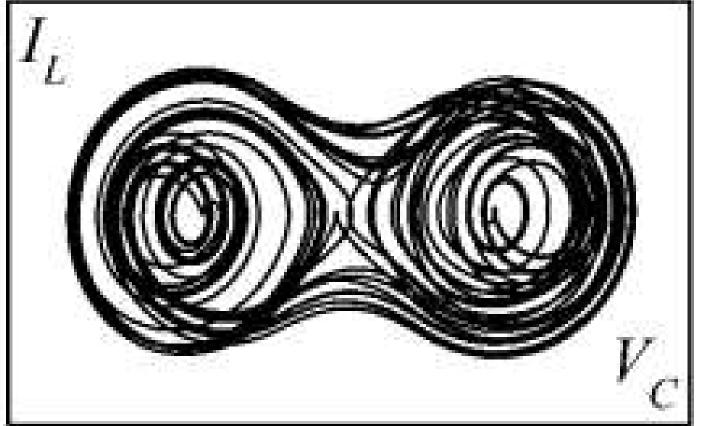
iL vs Vc (Period-2 Vc=142mv)



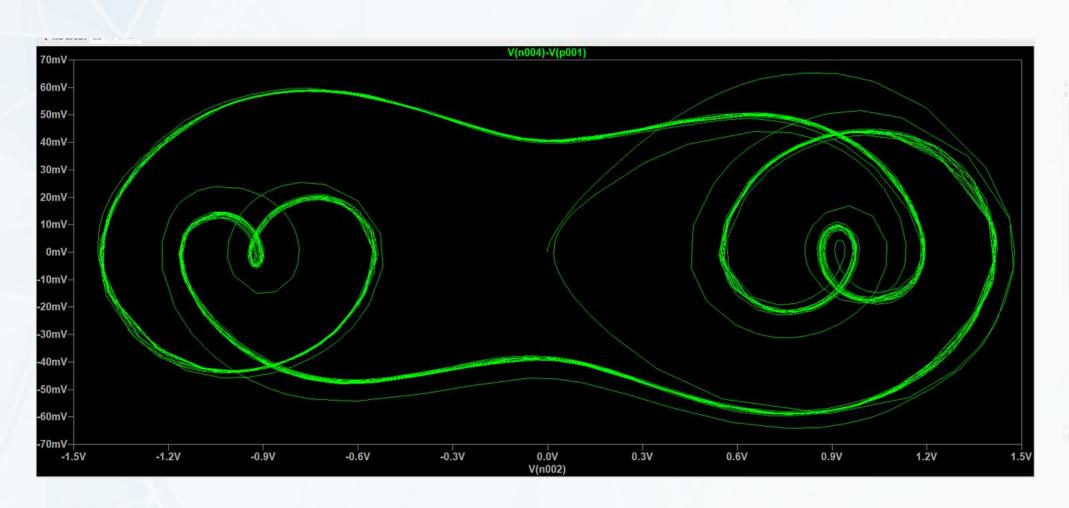


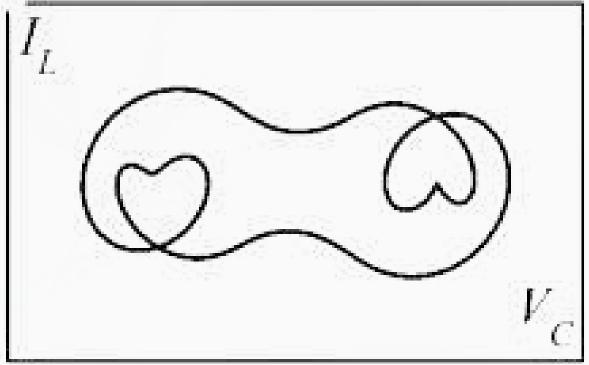
iL vs Vc (Chaos at 143mv)





iL vs Vc (Period-5 at 169mv)





Thank You and Good Night

Wait A minute...

In case I don't see ou on 31st ...

