

RENEWABLE ENERGY TECHNOLOGY

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Solar Thermal Power Technologies

Solar Thermal Electricity

Concentration Technologies

Currently four key solar thermal power technologies

Large Scale ($> 10\text{MW}_e$)



Linear Fresnel



Solar Tower

Modular ($< 50\text{kW}_e$)



Parabolic Dish



Parabolic Trough

Key Solar Technologies

- Each solar collector technology has its own specific range of practically achievable concentration ratios
- As such, each technology is adapted to one or more types of temperature range and thus power generation cycles

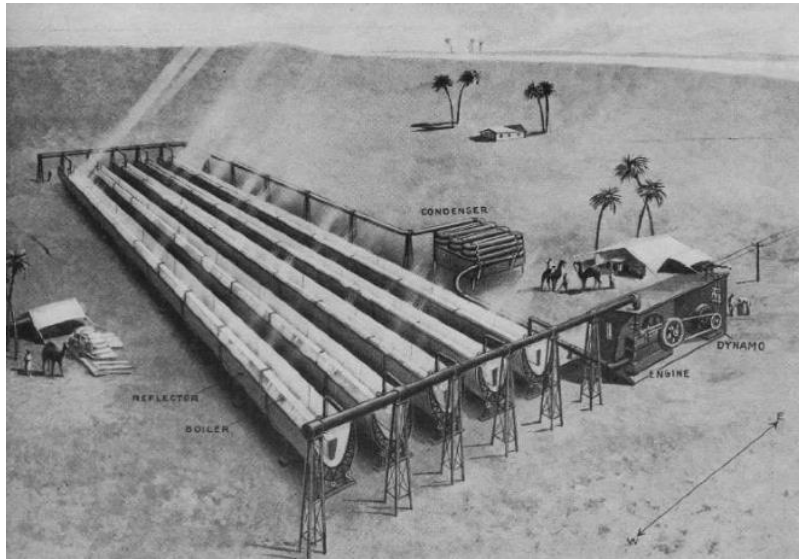
	Concentration	Tracking	Focal Spot	Temperatures
Linear Fresnel	15 – 60	One-Axis	Line	< 500°C
Parabolic Trough	30 - 100	One-Axis	Line	< 600°C
Heliostat Field	500 - 1'000	Two-Axis	Point	< 1200°C
Parabolic Dish	1'000 - 10'000	Two-Axis	Point	< 750°C

Data Source: C. Philibert, 2005

- Other technologies do exist, but are significantly less developed

Early Attempts at Solar Power

- First power producing solar thermal power plant was built in Egypt in 1913, using parabolic trough technology



- First patent deposited in 1907
- Steam production to drive a 40 kW reciprocating steam engine
- Payback time of 2 years against coal from England at 13 \$/ton



Image Source: Wikipedia, 2012

SEGS Power Plants

- First modern solar thermal power plants, the Solar Energy Generating Systems (or SEGS) were built in California in the 1980s
- Initial built to hedge against high oil/gas prices after the oil crises of the 1970s



SEGS 3-7, Kramer Jct.



Parabolic Troughs

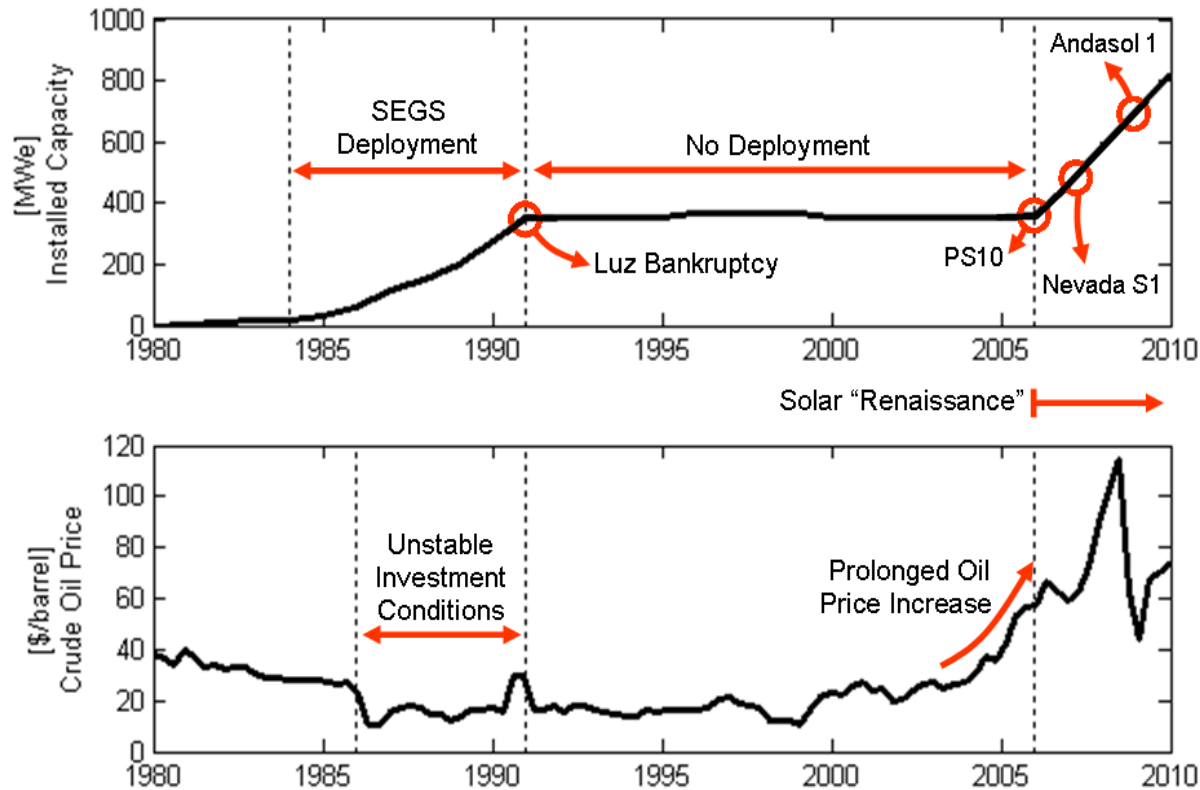


Mirror Washing



Recent CSP Deployment

- Solar thermal power development began in the 1980s with the SEGS. A new “solar renaissance” started in 2006 with new power plants



Spanish Solar Renaissance

- In 2004 a royal decree equalized conditions for CSP and PV plants
- Feed-in tariffs for solar energy were guaranteed, removing some economic barriers to the deployment of solar thermal technology

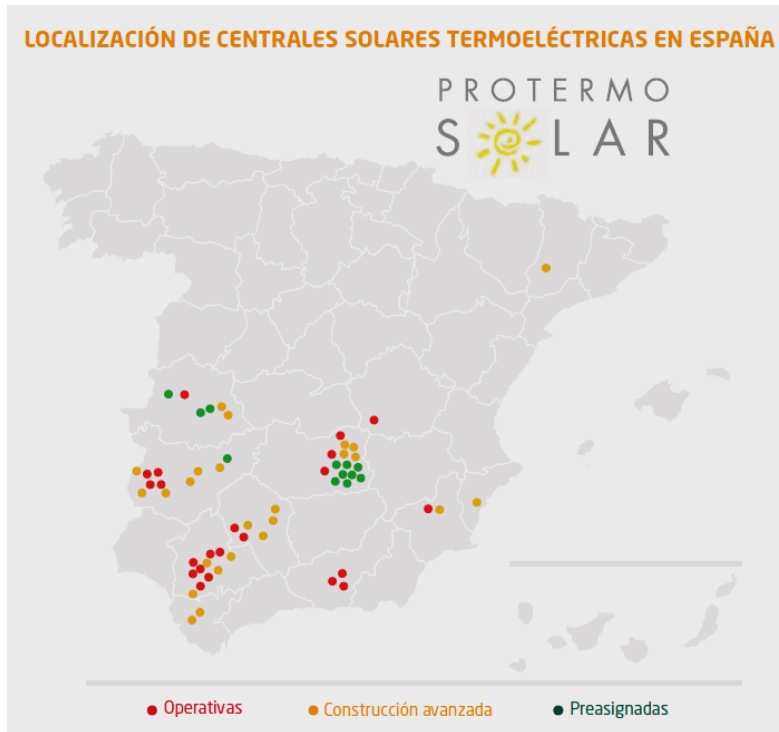


Image Source: Protermosolar, 2011

- By early 2012, over 1000 MW of solar thermal power had been deployed
- Another 1200 MW are currently under construction
- Over 90% of all CSP plants built are of the parabolic trough type

Commercial Solar Plants

- Commercial solar thermal power plants in Spain:

Andasol 1, 2 & 3



PS 10 & 20

Puerto Errado II



Solnova 1, 2 & 4



Solar Thermal Electricity

Parabolic Trough Power Plants

Parabolic Trough Plants

- Over 90% of all installed solar thermal power plants are based around the use of parabolic troughs with Rankine-cycles
- The technology was well-proven, making it easier to obtain funding when the second wave of CSP construction started
- Continuous operation of the SEGS plants since 1984
- However, limited innovation in commercial plants...



Types of Trough Plants

Two main types of parabolic trough plant have emerged:

- 'SEGS-type': daytime-peaking, no storage
- 'Andasol-type': day load and evening peak, with storage

Power-plants based around standard steam-cycle technology

- Compatible temperature levels between solar collector and power block
- Lower risk: well understood technology

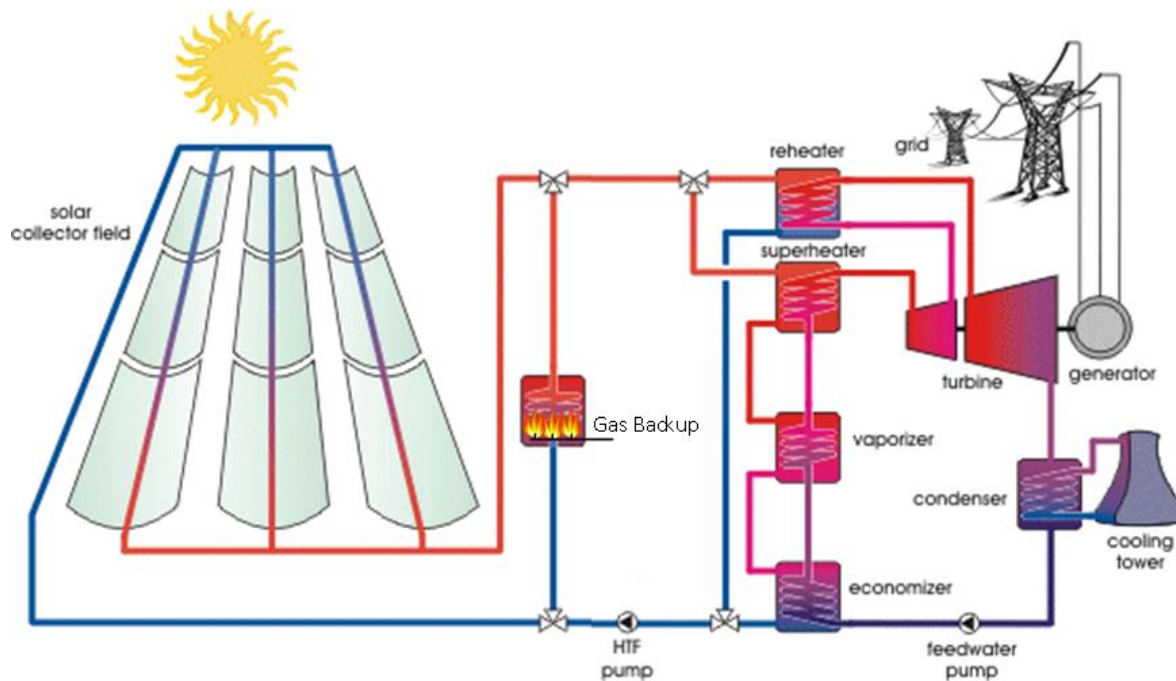
Plant design strongly affect by local regulation and incentive measures:

- USA: Loan guarantees and tax credits
- Spain:
 - ✓ Limited to 50MW power block
 - ✓ Limited to 13% fossil co-firing



SEGS – Type Power Plant

- Designed primarily to meet midday peak electricity demands
- Reheat steam cycle used to allow higher cycle efficiency at the low steam temperatures
- Operating temperatures limited by heat transfer fluid



Thermal Oil HTF (heat Transfer fluid)-System

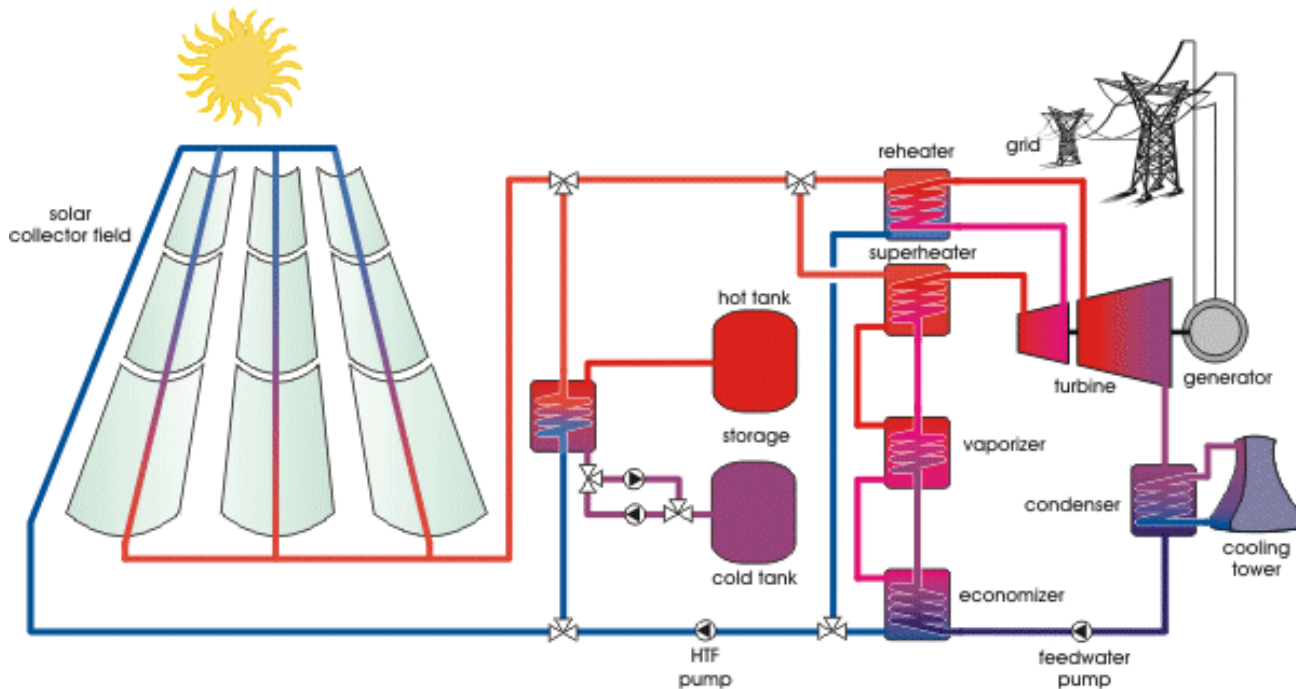
- Medium: Therminol-72
- Thermal Stability: 400°C

Power Block

- Reheat Rankine-cycle
- Steam Temperature: 390°C
- Steam Pressure: 100 bar

Andasol – Type Power Plant

- Designed to meet two daily peaks, midday and early evening
- Thermal energy storage tanks used to harness extra energy during daily hours, allowing production to be extended in the evening
- Larger solar field required to charge storage tanks



Molten-Salt Storage

Medium: $\text{NaNO}_3\text{-KNO}_3$

Thermal Stability: 580°C

Power Block

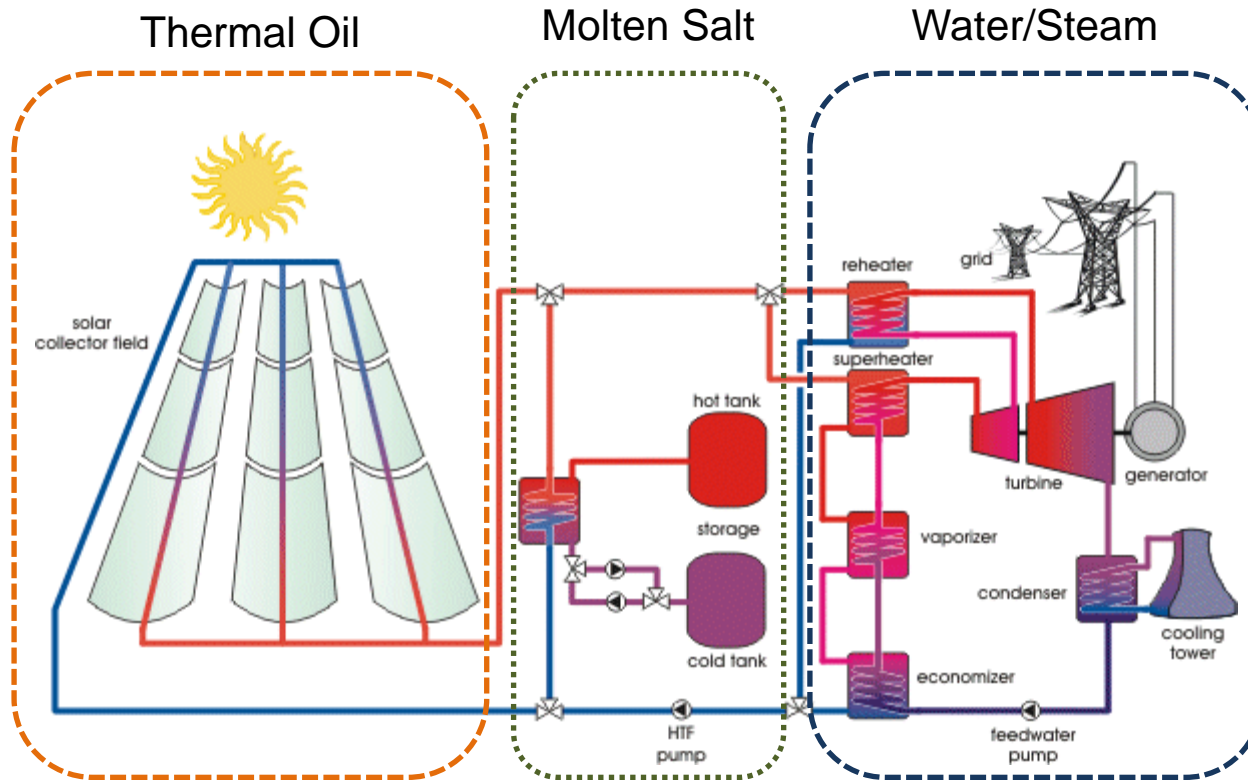
Reheat Rankine-cycle

Steam Temperature: 390°C

Steam Pressure: 100 bar

Molten – Salt Storage System

- Thermal energy storage based on molten salts adds complexity to the system, as three separate fluid loops are required



Thermal Oil

- Good heat transfer
- Low freezing point
- No phase-change

Molten Salt

- High heat capacity
- Pre-available product
- Inexpensive
- Chemically inert

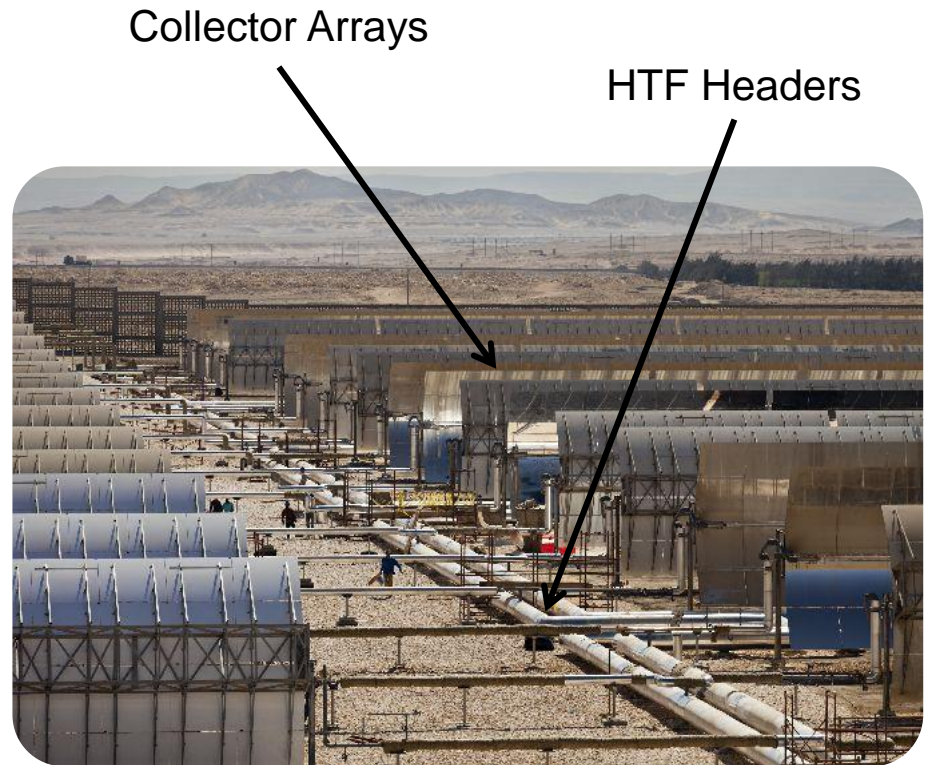
Parabolic Trough Plants

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Molten Salt
Storage Tanks

Power Block

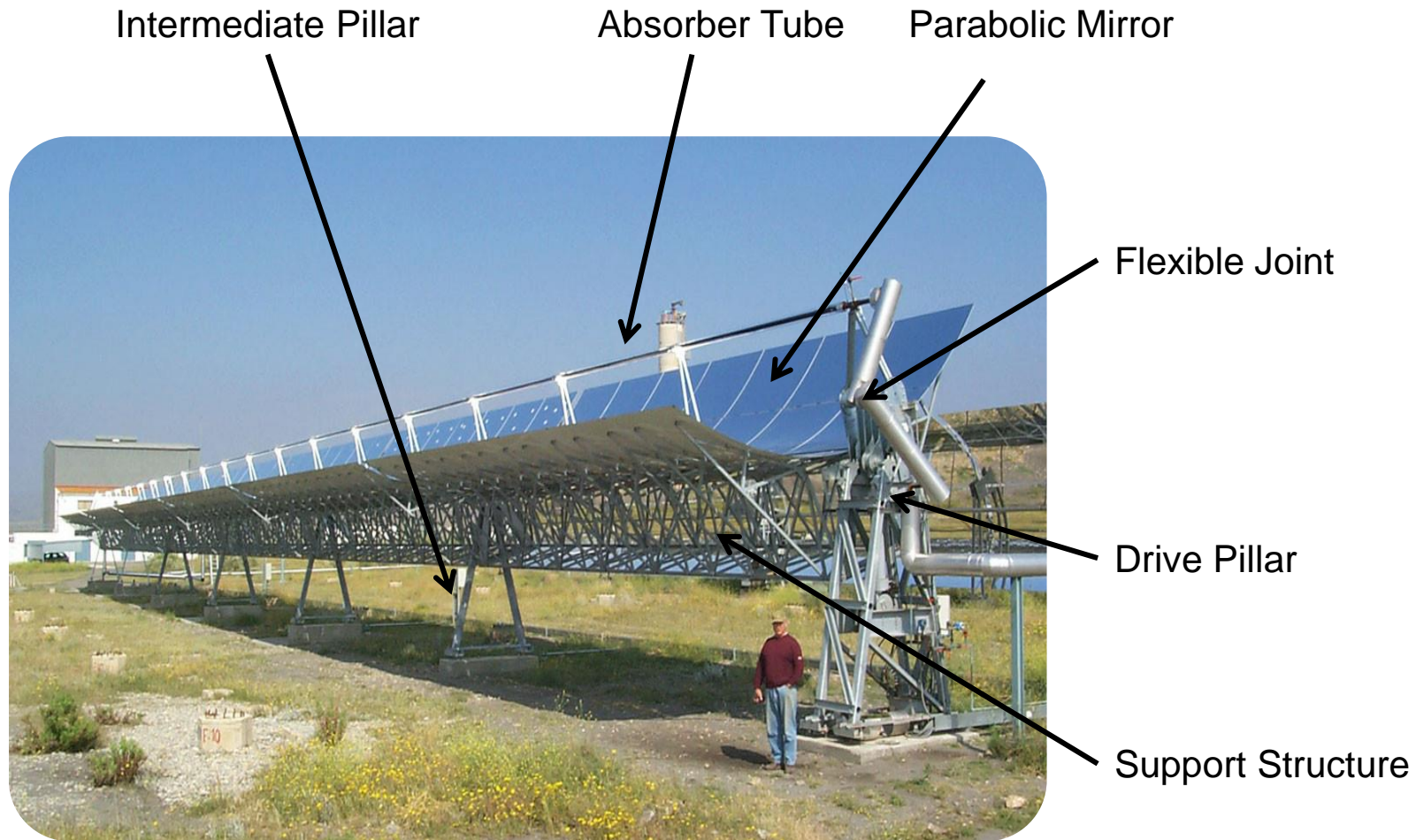


Collector Arrays

HTF Headers

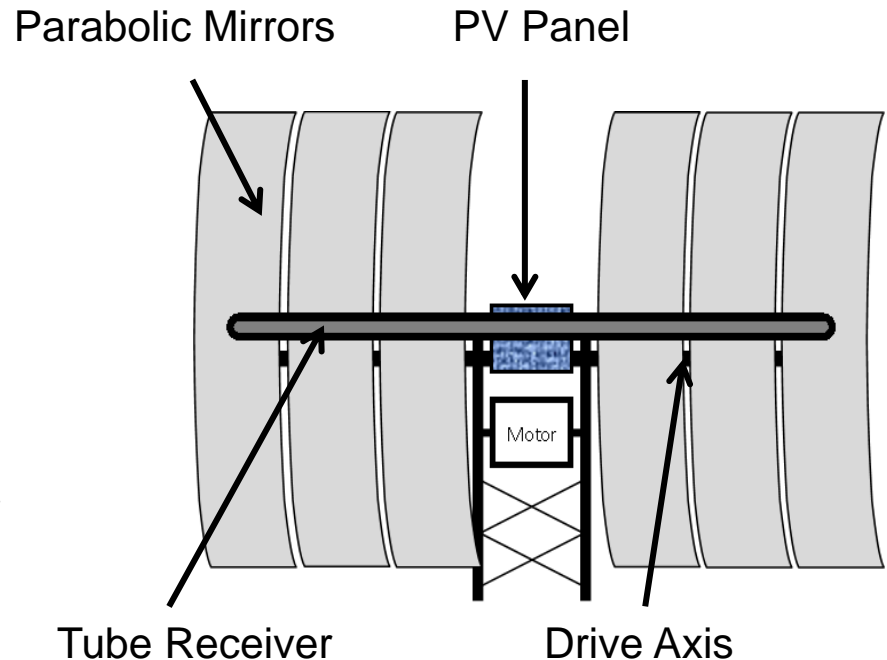
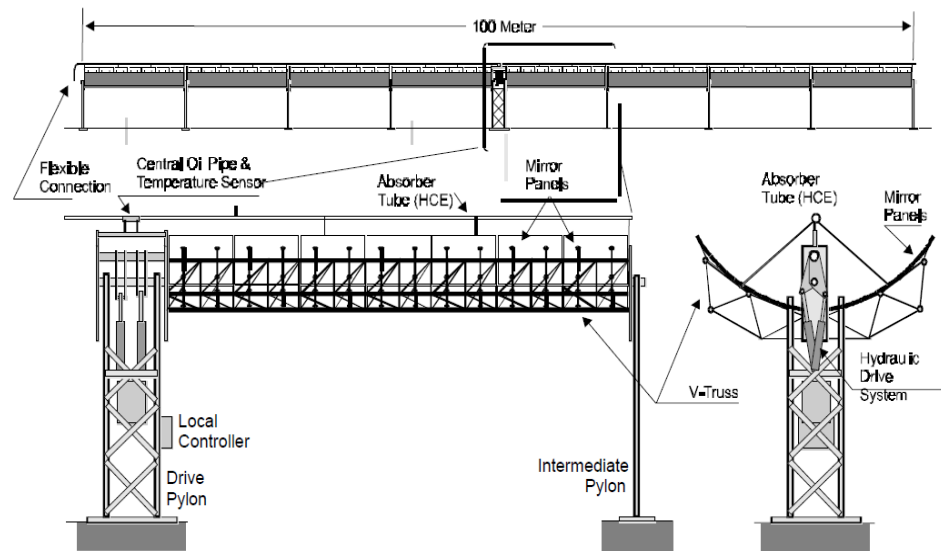
Parabolic Trough Collector

- A large number of different parabolic collector designs have been proposed but all share a similar structure



Parabolic Trough Collector

- A large number of different parabolic collector designs have been proposed but all share a similar structure
- The central drive pillar provides tracking power and control for the entire solar collector assembly



Output of a Trough Collector

The net energy supplied by a given parabolic trough collector is a function of a number of factors, and can be defined by:

$$\dot{Q}_{tr}^+ = A_{tr} I_o \cdot \varepsilon_{surf} \varepsilon_{cos} \cdot IAM \cdot (1 - f_{shd}) (1 - f_{end})$$

\dot{Q}_{tr} : thermal power [W]

A_{tr} : trough aperture area [m²]

I_o : incident beam radiation [W/m²]

ε_{surf} : surface effectiveness [-]

ε_{cos} : cosine effectiveness [-]

f_{shd} : shadowing factor [-]

IAM: incidence angle modifier [-]

f_{end} : end-loss factor [-]

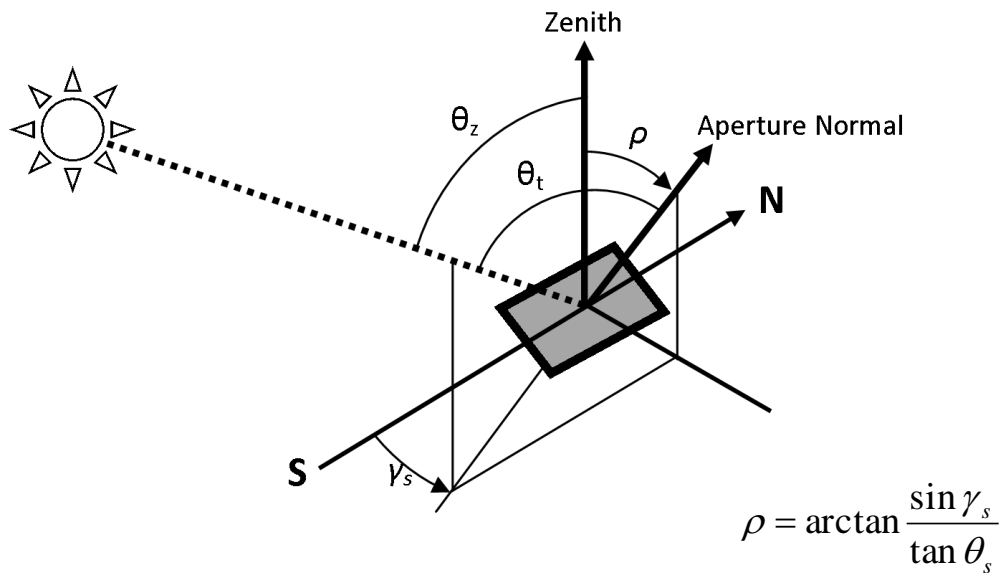


Cosine Effectiveness

- Commercial parabolic troughs employ NS-single-axis tracking, resulting in non-zero incidence angles

$$\varepsilon_{\cos} = \cos \theta_t \quad \text{with} \quad \cos \theta_t = \sqrt{1 - \cos^2 \theta_s \cos^2 \gamma_s}$$

- North-south aligned parabolic troughs give flat efficiency in summer



θ_t : Trough Incidence Angle

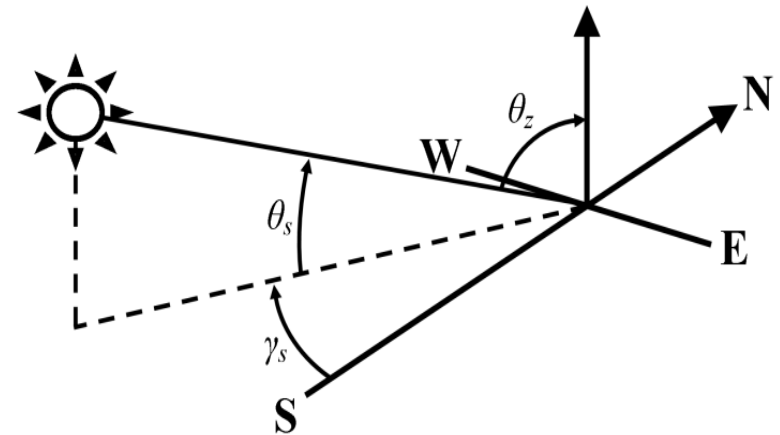


Image Source: J. Spelling, 2011

Incidence Angle Modifier

- In addition to the basic cosine-effect, other optical losses occur when the incidence angle is non-zero
- The incidence angle modifier (IAM) considers the optical path-length

$$\text{IAM} = 1 - a_1 \frac{\theta_t}{\cos \theta_t} + a_2 \frac{\theta_t^2}{\cos \theta_t}$$

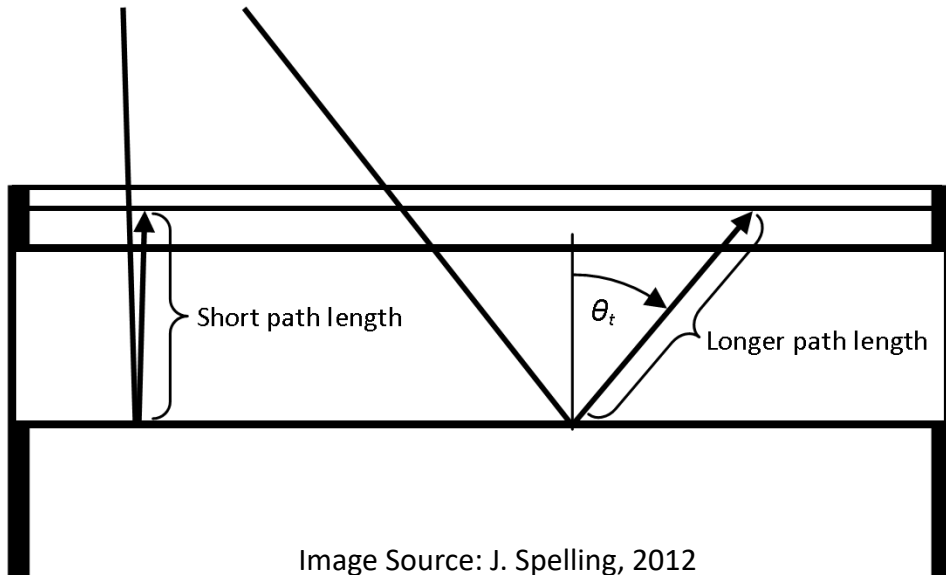
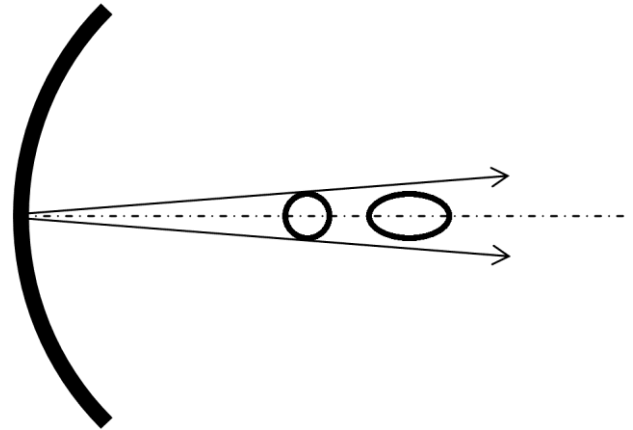


Image Source: J. Spelling, 2012



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LS-2 Collector	Value
a_1	0.0159
a_2	0.1405

Data Source: Dudley *et al.*, 1994

Unique for each collector design!

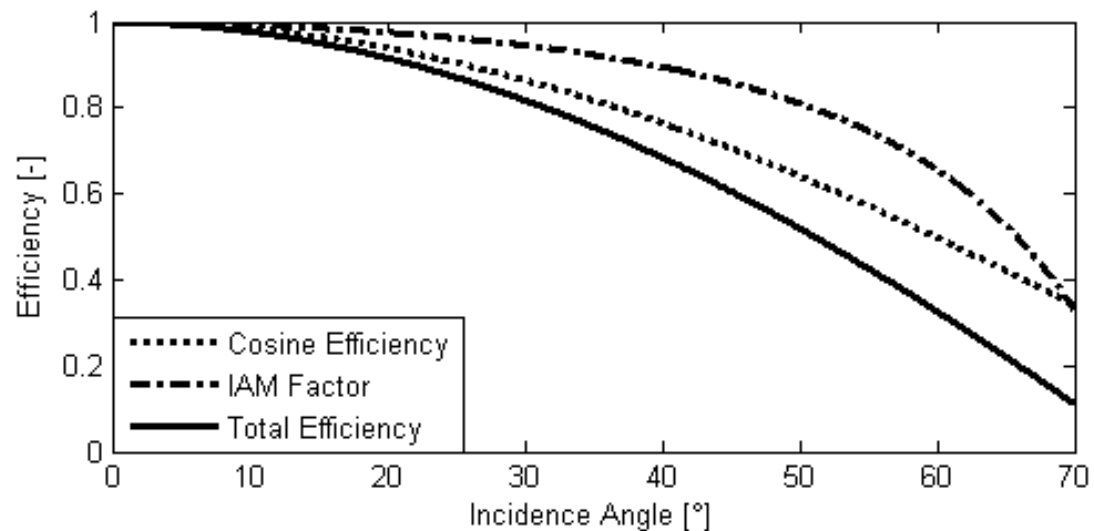


Image Source: J. Spelling, 2012

Tube – End Losses

- Tube-end losses take into account the fact that the collector has a finite length, leading to a certain inactive length of the collector

$$f_{end} = \frac{l_{focus} \tan \theta_t}{l_{trough}}$$

l_{focus} : Trough Focal Length

l_{trough} : Entire SCA length

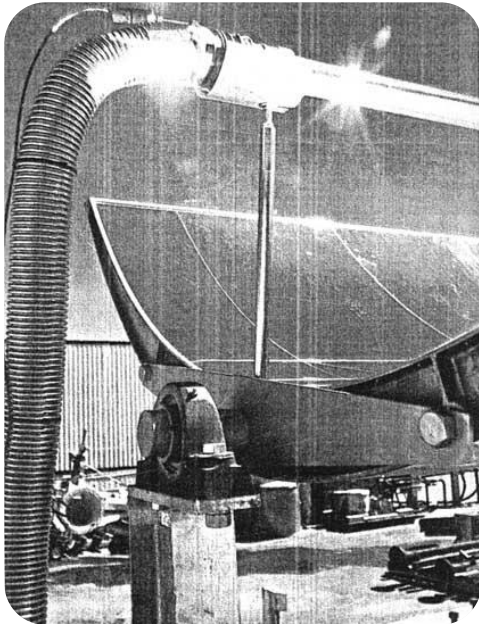


Image Source: Stine, 1987

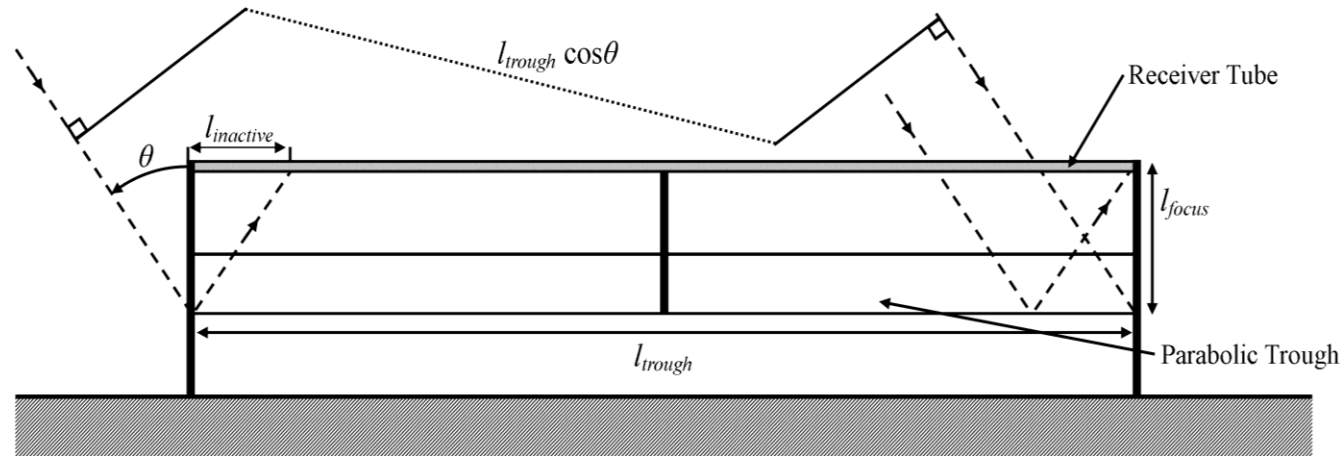


Image Source: J. Spelling, 2011

Shadowing Losses

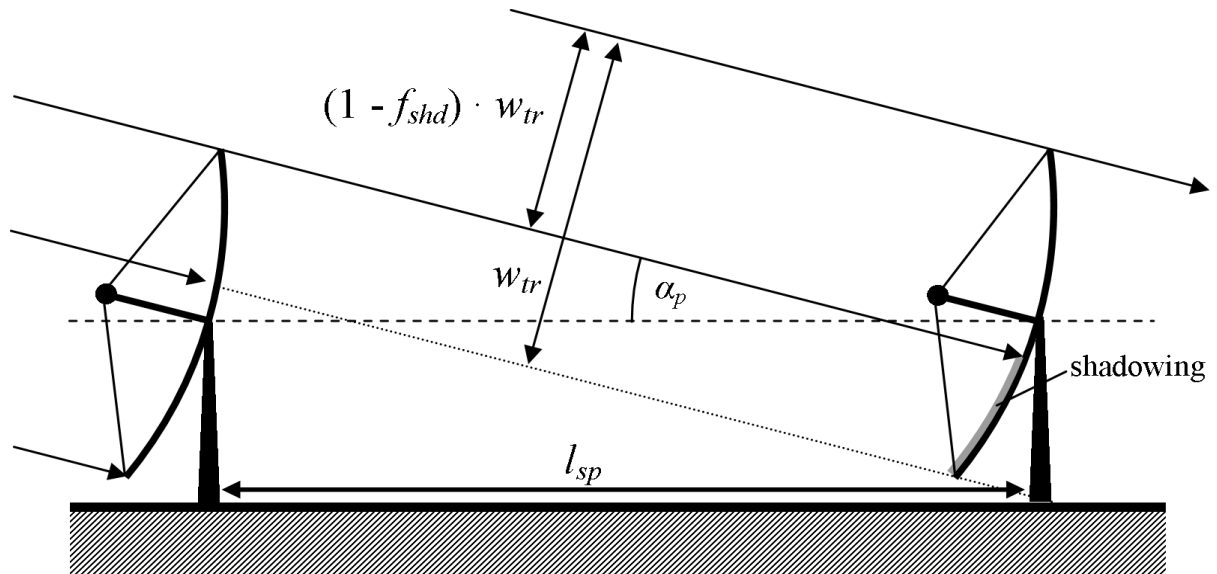
- As the collectors are not operated in isolation, the effect of shadowing between rows needs to be taken into consideration:

$$f_{shd} = 1 - \frac{l_{sp}}{w_{tr}} \frac{\cos \theta_z}{\cos \theta_t}$$

l_{sp} : Spacing between trough rows

w_{tr} : Parabolic Trough Width

θ_z : Solar Zenith Angle



MATHMETICAL PROBLEMS

1. A solar collector system consists of a receiver with surface area of 0.08 m^2 . The aperture area of the collector is 9 m^2 . The solar radiation from the aperture is 700 W/m^2 and the radiation received by the receiver is 1500 W/m^2 . (Emissivity, $\varepsilon=0.5$ and Absorptivity, $\alpha=0.7$) Calculate-

1. Geometric and optical concentration ratio of the system.
2. Maximum temperature that can be achieved on the receiver surface.
3. If the surface temperature is 150°C , calculate the useful energy extracted from the receiver.

2. A parabolic through solar thermal power plant (located at 23°S , 18°W) has the following dimensions

Parabolic Trough Width= 1.25m

Through Focal length= 1.5m

Entire length of the collector aperture= 50m

Total aperture area= 128 m^2

Separation between two rows= 5m

Surface effectiveness= 0.08

Calculate the thermal power supplied by the parabolic through collector at solar noon on 11th February if the incident beam radiation is 500 W/m^2 . $A_1=.0159$, $A_2=.1405$

Renewable Energy Technology

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