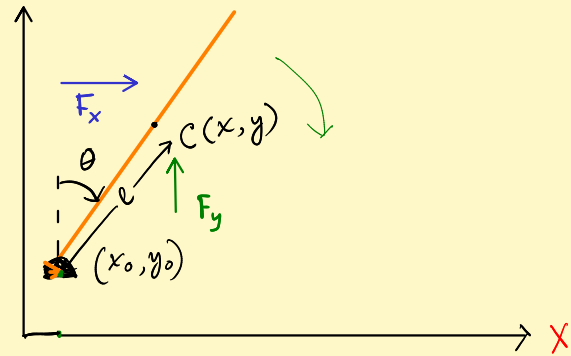


● Inverted Pendulum

A rod of mass m whose centre of mass is located at (x, y) and a end is situated at (x_0, y_0) which is $f(t)$. The rod is free to move the pivot point.



If moment of inertia about the com is I_c . And force acting on the pivot point is F_x & F_y along x and y direction respectively. Then net torque on the system about c ,

▣ **Newtonian approach:**

$$I_c \ddot{\theta} = F_y l \sin \theta - F_x l \cos \theta \quad \text{--- (1)}$$

$$\text{Now, } F_x = m\ddot{x} \quad ; \quad F_y = m\ddot{y} - mg \quad \text{--- (2)}$$

$$\begin{aligned} x &= x_0 + l \sin \theta \\ \Rightarrow \dot{x} &= \dot{x}_0 + l \cos \theta \dot{\theta} \\ \Rightarrow \ddot{x} &= \ddot{x}_0 + l \cos \theta \ddot{\theta} - l \dot{\theta}^2 \sin \theta \quad \dots (3.a) \end{aligned}$$

$$\begin{aligned} y &= y_0 + l \cos \theta \\ \Rightarrow \dot{y} &= \dot{y}_0 - l \sin \theta \dot{\theta} \\ \Rightarrow \ddot{y} &= \ddot{y}_0 - l \sin \theta \ddot{\theta} - l \dot{\theta}^2 \cos \theta \quad \dots (3.b) \end{aligned}$$

Now putting the values in eq (1) from (2) & (3).

$$I_c \ddot{\theta} = ml \sin \theta (\ddot{y}_0 - l \sin \theta \ddot{\theta} - l \dot{\theta}^2 \cos \theta - g) - ml \cos \theta (\ddot{x}_0 + l \cos \theta \ddot{\theta} - l \dot{\theta}^2 \sin \theta)$$

cancel each-other

$$\Rightarrow I_c \ddot{\theta} = ml (\sin \theta \ddot{y}_0 - \cos \theta \ddot{x}_0) - ml^2 \ddot{\theta} (\sin^2 \theta + \cos^2 \theta) - mlg \sin \theta$$

$$\Rightarrow (I_c + ml^2) \ddot{\theta} = ml \sin \theta (\ddot{y}_0 + g) - ml \cos \theta \ddot{x}_0$$

↓
Moment of inertia of the rod about the pivot point

$$\boxed{\ddot{\theta} = \left(\frac{ml}{I} \right) ((\ddot{y}_0 + g) \sin \theta - \ddot{x}_0 \cos \theta)} \quad [I = I_c + ml^2]$$

• Lagrangian approach :-

$$T = \frac{1}{2} m (\dot{x}^2 + \dot{y}^2)$$

$$= \frac{1}{2} m (\dot{x}_0^2 + 2x_0 l \cos \theta \ddot{\theta} + l^2 \ddot{\theta}^2 + \dot{y}_0^2 + 2y_0 l \sin \theta \ddot{\theta})$$

$$V = mgy = mg(y_0 + l \cos \theta)$$

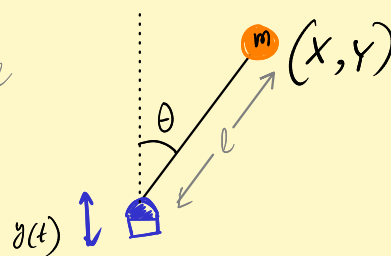
$\mathcal{L} = T - V$ and use EL eqⁿ for all generalised coordinate.

$$\begin{aligned} x &= x_0 + l \sin \theta & \dot{x} &= \dot{x}_0 + l \cos \theta \dot{\theta} \\ y &= y_0 + l \cos \theta & \dot{y} &= \dot{y}_0 - l \sin \theta \dot{\theta} \end{aligned}$$

• Case 1: Vertically driven ^{rod have mass = 0} simple pendulum:

Given, $y(t) = A \sin(\omega_d t)$ $A \ll l$

$$\begin{aligned} X &= l \sin \theta & Y &= l \cos \theta + y(t) \\ \Rightarrow \dot{X} &= l \cos \theta \dot{\theta} & \Rightarrow \dot{Y} &= -l \sin \theta \dot{\theta} + A \omega_d \cos(\omega_d t) \end{aligned}$$



$$T = \frac{1}{2} m (\dot{X}^2 + \dot{Y}^2) = \frac{1}{2} m (l^2 \cos^2 \theta \dot{\theta}^2 + l^2 \sin^2 \theta \dot{\theta}^2 - 2A\omega_d l \sin \theta \cos(\omega_d t) \dot{\theta} + A^2 \omega_d^2 \cos^2(\omega_d t))$$

$$\Rightarrow T = \frac{1}{2} m (l^2 \dot{\theta}^2 - 2A\omega_d l \sin \theta \cos(\omega_d t) \dot{\theta} + A^2 \omega_d^2 \cos^2(\omega_d t))$$

$$V = mgy = mg(l \cos \theta + A \sin(\omega_d t))$$

$$\mathcal{L} = T - V \quad \& \quad \boxed{\frac{d}{dt} \left(\frac{\partial \mathcal{L}}{\partial \dot{\theta}} \right) = \frac{\partial \mathcal{L}}{\partial \theta}}$$

$$\frac{\partial \mathcal{L}}{\partial \dot{\theta}} = m(l^2 \dot{\theta} - A\omega_d l \sin \theta \cos(\omega_d t))$$

$$\Rightarrow \frac{d}{dt} \left(\frac{\partial \mathcal{L}}{\partial \dot{\theta}} \right) = m(l^2 \ddot{\theta} - A\omega_d l \cos \theta \cos(\omega_d t) \dot{\theta} + A\omega_d^2 l \sin \theta \sin(\omega_d t))$$

$$\frac{\partial \mathcal{L}}{\partial \theta} = -mA\omega l \cos\theta \cos(\omega_d t) \dot{\theta} + mgl \sin\theta$$

$$\therefore m l^2 \ddot{\theta} = mgl \sin\theta - A m \omega_d^2 l \sin\theta \sin(\omega_d t)$$

$$\ddot{\theta} = (g - A \omega_d^2 \sin(\omega_d t)) \left(\frac{\sin\theta}{l} \right)$$

This eqⁿ can be derived also from Newtonian approach,

$$I \ddot{\theta} = m l (\sin\theta (\ddot{y}_0 + g) - \cos\theta \ddot{x}_0)$$

$$\text{Here } I = m l^2 \text{ \& } \ddot{y}_0 = -A \omega_d^2 \sin(\omega_d t) \text{ \& } \underline{\ddot{x}_0 = 0}$$

$$\ddot{\theta} m l^2 = m l \sin\theta (g - A \omega_d^2 \sin(\omega_d t))$$

$$\Rightarrow \ddot{\theta} = \left(\frac{\sin\theta}{l} \right) (g - A \omega_d^2 \sin(\omega_d t))$$

• Non linear eqⁿ so we have solve it using numerically.

• Simultaneous eqⁿ: $\dot{\theta} = \omega \quad \dot{\omega} = [g - A \omega_d^2 \sin(\omega_d t)] \left(\frac{\sin\theta}{l} \right)$

We can get some idea using small angle approximation,
 $\sin\theta \approx \theta$

$$\text{So, } \ddot{\theta} = \frac{\theta}{l} (g - A \omega_d^2 \sin(\omega_d t))$$

$$\theta = e^{\lambda t}$$

$$\Rightarrow \lambda^2 = \frac{1}{l} (g - A \omega_d^2 \sin(\omega_d t))$$

$$\therefore \lambda = \pm \sqrt{\frac{1}{l} (g - A \omega_d^2 \sin(\omega_d t))}$$

$$\theta(t) = c_1 e^{\lambda_1 t} + c_2 e^{\lambda_2 t}$$

$$\begin{aligned}
 m l^2 \ddot{\theta} &= m g l \sin \theta - A m \omega_d^2 l \sin \theta \sin(\omega_d t) \\
 &= - \left[\underbrace{A \omega_d^2 \sin(\omega_d t)}_{\text{yellow}} - \underbrace{g}_{\text{green}} \right] m l \sin \theta
 \end{aligned}$$

If $A \omega_d^2 > g \rightarrow$ System will ^{try} to stabilize itself

● Case 2: only horizontal driving force:-

$$y_0 = 0; \dot{y}_0 = 0 \quad \& \quad x_0 = A \sin(\omega_d t) \Rightarrow \ddot{x}_0 = -A \omega_d^2 \sin(\omega_d t) \quad \ddot{\theta} = \left(\frac{m l}{I} \right) ((\ddot{y}_0 + g) \sin \theta - \ddot{x}_0 \cos \theta)$$

So, eqⁿ of motion,

$$\ddot{\theta} = \frac{m l}{m l^2} (g \sin \theta + A \omega_d^2 \sin(\omega_d t) \cos \theta)$$

$$\ddot{\theta} = \frac{1}{l} (g \sin \theta + A \omega_d^2 \sin(\omega_d t) \cos \theta)$$