

Advanced Dynamics - Assignment Report

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1 Assignment 1

1 WHEEL ROLLING WITHOUT SLIPPING ON A 2D TRACK

A thin wheel of radius R rolls without slipping on a track on the $x_1 - x_2$ plane, defined by $x_2 = f(x_1)$. The wheel plane stays vertical and tangent to such track at the contact point P . Denote with α the angle the disk plane forms with the x_2 axis, and with ϕ the rotation of the disk about its axis \mathbf{e}_ϕ . The position of the center of the disk C is indicated by x_1^C , x_2^C and x_3^C . Assume a set of generalized coordinate $\mathbf{q} = [x_1^C \ x_2^C \ x_3^C \ \alpha \ \phi]$.

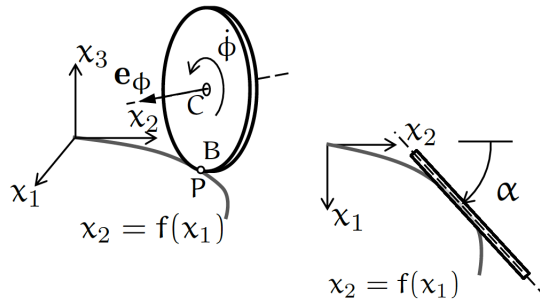


Figure 1.1: Wheel rolling without slipping on a track.

1. State all the constraints acting on the disk.
2. Determine whether the constraints are holonomic or non-holonomic.

Figure 1: Task 1.1

1.1

The list of constraints looks as follows:

1. The wheel always stays vertical (plane parallel to x_3)
This is a holonomic constraint: $f = \theta = 0$ where θ denotes the angle between the disk and x_3
2. x_2 follows a fixed trajectory, given x_1 (and vice versa):
 $f_2 : x_2 = f(x_1) \Rightarrow x_2 - f(x_1) = 0 \Rightarrow$ holonomic.
3. α is the angle between the trajectory and the x_2 axis:
 $\alpha = \frac{\pi}{2} - \frac{\partial f(x_1)}{\partial x_1}$ or written differently:
 $f(\alpha, x_1) = \alpha - \frac{\pi}{2} + \frac{\partial f(x_1)}{\partial x_1} = 0 \Rightarrow$ holonomic
4. Rolling without slipping:
 $v_B = 0 \Rightarrow v_C + \omega \times R_{CB} = 0$ with $\omega = \dot{\alpha}\mathbf{e}_3 + \dot{\phi}\mathbf{e}_\phi$
Seems to be non-holonomic at first glance
5. The disk does not leave the ground:
 $x_3^C - R = 0$ aka the x_3 component of the center of mass is R.
This is holonomic as well

So far we have a 3D system (6 DoF) and 4 holonomic constraints and 1 non-holonomic constraint.

4. Check for integrability: To Do

$$v_B = 0 \Rightarrow v_C + \omega \times R_{CB} = 0 \quad (1)$$

Plugging in $\dot{x}_1^C, \dot{x}_2^C, \omega = \dot{\alpha}\mathbf{e}_3 + \dot{\phi}\mathbf{e}_\phi$ and $R_{CB} = [0, 0, -R]^T$:

$$\dot{x}_1^C \mathbf{e}_1 + \dot{x}_2^C \mathbf{e}_2 - R\dot{\phi} \cos \alpha \mathbf{e}_2 - R\dot{\phi} \sin \alpha \mathbf{e}_1 = 0 \quad (2)$$

Considering the part in \mathbf{e}_1 direction:

$$\dot{x}_1^C - R\dot{\phi} \sin \alpha = 0 \quad (3)$$

Now if we reformulate this in the linear velocity form:

$$\sum_{i=1}^n a_i(\mathbf{q}, t) \dot{q}_i + b(\mathbf{q}, t) \text{ with } \mathbf{q} = [x_1^C, x_2^C, x_3^C, \alpha, \phi] \quad (4)$$

We get

$$\begin{aligned}
a_1 &= 1, \quad a_5 = -R \sin(\alpha), b = 0 \\
\Rightarrow \frac{\partial(Cb)}{\partial q_1} &= \frac{\partial(Ca_1)}{\partial t} \Rightarrow \frac{\partial C}{\partial t} = 0 \Rightarrow C \neq C(t) \\
\Rightarrow \frac{\partial(Ca_1)}{\partial q_5} &= \frac{\partial(Ca_5)}{\partial q_1} \Rightarrow \frac{\partial(C)}{\partial \phi} = \frac{\partial(-CR \sin(\alpha))}{\partial x_1^C} \\
\Rightarrow \frac{\partial(C)}{\partial \phi} &= -R \sin(\alpha) \frac{\partial(C)}{\partial x_1^C}
\end{aligned} \tag{5}$$

Perhaps we can show directly that the constraint can be written in the solution form:

$$\dot{x}_1^C \mathbf{e}_1 - R \dot{\phi} \sin \alpha \mathbf{e}_1 = \sum_{i=1}^n \frac{\partial f(\mathbf{q}, t)}{x_1^C} \dot{x}_1^C + \frac{\partial f(\mathbf{q}, t)}{\phi} \dot{\phi} \tag{6}$$

1.2

See section 1.1

1.3

3. Determine the degrees of freedom of the system.

Figure 2: Task 1.1.3

As we have a 3D body with 6 generalized coordinates (here 5 are given, already considering constraint 1) and 5 holonomic constraints. We get a total of $6 - 5 = 1$ degree of freedom. That could for instance be the rotation of the wheel around \mathbf{e}_ϕ while all the other generalized coordinates follow accordingly.

2 Assignment 2

3 Assignment 3

3.1

Lagrange equations:

$$\frac{L^2 M \ddot{\phi}}{3} + \frac{L^2 m \ddot{\phi}}{3} - \frac{L^2 m \ddot{\phi} \cos(\beta)^2}{3} + \frac{LMg \cos(\beta) \sin(\phi)}{2} + \frac{L^2 M \Omega^2 \sin(\beta) \sin(\phi)}{2} + \frac{L^2 M \Omega^2 \cos(\beta) \sin(\beta) \sin(\phi)}{4} - \frac{L^2 M \Omega^2 \cos(\beta)^2 \cos(\phi) \sin(\phi)}{6} = 0 \quad (7)$$

3.2

This can be reformulated for the differential equation of $\ddot{\phi}$:

$$\ddot{\phi} \left(\frac{L^2(M+m)}{3} - \frac{L^2 m \cos(\beta)^2}{3} \right) + \sin(\phi) \left(\frac{LMg \cos(\beta)}{2} + \frac{L^2 M \Omega^2 \sin(\beta)}{2} + \frac{L^2 M \Omega^2 \cos(\beta) \sin(\beta)}{4} \right) - \frac{L^2 M \Omega^2 \cos(\beta)^2 \cos(\phi) \sin(\phi)}{6} = 0 \quad (8)$$

3.3

Equation of motion for the case that the rotation around the vertical bar is not constant:

$$\frac{L^2 \ddot{\theta} \left(16M + 16m + 12M \cos(\beta) + 12m \cos(\beta) + 2M \cos(\beta)^2 + 4m \cos(\beta)^2 - 2M \cos(\beta)^2 \cos(\phi)^2 \right)}{12} + \frac{L^2 \ddot{\theta} (12M \cos(\phi) \sin(\beta) + 6M \cos(\beta) \cos(\phi) \sin(\beta))}{12} \quad (9)$$

3.4

After matlab integration of the work per area we get for the work

$W =$

$$\frac{L^4 \left(c \left(\dot{\theta} + \dot{\phi} \sin(\beta) \right)^2 \left(\cos(\phi)^2 - 1 \right) - c \left(\dot{\phi} \cos(\phi) + \dot{\theta} \cos(\phi) \sin(\beta) \right)^2 + c \dot{\phi}^2 \cos(\beta)^2 \left(\cos(\phi)^2 - 1 \right) \right)}{3} - L \left(Lc \left(L\dot{\theta} + L\dot{\theta} \cos(\beta) \right)^2 + \frac{L^3 c \dot{\theta}^2 \cos(\beta)^2}{12} \right) - L^3 c \left(L\dot{\theta} + L\dot{\theta} \cos(\beta) \right) \left(\dot{\phi} \cos(\phi) + \dot{\theta} \cos(\phi) \sin(\beta) \right) \quad (10)$$

3.5

Reminder- The potential energy looks as follows:

$$\frac{LMg(2\sin(\beta) - \cos(\beta)\cos(\phi))}{2} \quad (11)$$

Case 1: Ω fixed:

The derivative of the potential energy w.r.t. θ is always 0.

$$\frac{\partial V}{\partial \theta} = 0 \quad (12)$$

The derivative of the potential energy w.r.t. ϕ not however:

$$\frac{\partial V}{\partial \phi} = \frac{LMg\cos(\beta)\sin(\phi)}{2} \quad (13)$$

Equation 12 is only 0 for $\phi = k*\pi$ which makes sense as in this configuration the square has a momentary velocity in horizontal direction which doesn't change the altitude of any body.

Case 2: $\dot{\theta}$ can change:

As V is independent of θ altogether the result is the same for this case.

The equation of motions are:

$$\begin{aligned} & \frac{L^2 M \ddot{\phi}}{3} + \frac{L^2 m \ddot{\phi}}{3} - \frac{L^2 m \ddot{\phi} \cos(\beta)^2}{3} + \frac{LMg\cos(\beta)\sin(\phi)}{2} + \frac{L^2 M \dot{\theta}^2 \sin(\beta)\sin(\phi)}{2} - \\ & \frac{L^2 M \dot{\theta}^2 \cos(\beta)^2 \cos(\phi)\sin(\phi)}{6} + \frac{L^2 M \dot{\theta}^2 \cos(\beta)\sin(\beta)\sin(\phi)}{4} \end{aligned} \quad \text{for } \phi \quad (14)$$

$$\begin{aligned} & \frac{L^2 \ddot{\theta}}{12} \left(16M + 16m + 12M\cos(\beta) + 12m\cos(\beta) + 2M\cos(\beta)^2 + 4m\cos(\beta)^2 - \right. \\ & \left. 2M\cos(\beta)^2\cos(\phi)^2 + 12M\cos(\phi)\sin(\beta) + 6M\cos(\beta)\cos(\phi)\sin(\beta) \right) \end{aligned} \quad \text{for } \theta \quad (15)$$

Plugging in $\phi = \dot{\phi} = \ddot{\phi} = 0$:

$$\begin{aligned}\phi : \\ 0 = 0\end{aligned}\tag{16}$$

$$\begin{aligned}\theta : \\ \frac{L^2\ddot{\theta}}{12} (16M + 16m + 12M \cos(\beta) + 12M \sin(\beta) + 12m \cos(\beta) + \\ 4m \cos(\beta)^2 + 6M \cos(\beta) \sin(\beta)) = 0 \Rightarrow \ddot{\theta} = 0\end{aligned}\tag{17}$$

As the coefficient of $\ddot{\theta}$ is constant, the angular acceleration of the vertical shaft has to be 0. This results in a constant angular velocity which recovers the case of a constant Ω .

3.6

4 Assignment 4