## **Assignment 3**

Consider the equations of motion

$$\frac{GJ}{L} \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix} \begin{Bmatrix} \theta_{X1} \\ \theta_{X2} \end{Bmatrix} + \frac{\rho LJ}{2} \begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix} \begin{Bmatrix} \ddot{\theta}_{X1} \\ \ddot{\theta}_{X2} \end{Bmatrix} = 0$$

for the end point rotations of a certain torsion bar of length L. Above, J is the second moment of area with respect to the axis of the bar (polar moment), G is the shear modulus, and  $\rho$  is the density of material. Derive the angular speeds and the corresponding modes of the free vibrations.

## **Solution template**

The set of ordinary differential equations as given by the principle of virtual work  $M\ddot{a} + Ka = 0$  consists of the inertia and stiffness parts. The symmetric mass matrix M and the stiffness matrix K depend on the structure. Angular speeds of the free vibrations are the eigenvalues of  $\Omega = \sqrt{M^{-1}K}$ . In practice, it is easier to calculate first the eigenvalues  $\Omega^2 = M^{-1}K$  as the eigenvalues of  $\Omega$  are the square roots of those for  $\Omega^2$  and the eigenvectors coincide.

In the present case, the matrices are

$$\mathbf{K} = \frac{GJ}{L} \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix} \text{ and } \mathbf{M} = \frac{\rho JL}{2} \begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix}.$$

Therefore

$$\mathbf{\Omega}^2 = \mathbf{M}^{-1}\mathbf{K} = \begin{bmatrix} & & & & \\ & & & & \\ & & & & \end{bmatrix}.$$

In the eigenvalue problem of matrix  $\bf A$ , the goal is to find all pairs  $(\lambda, {\bf x})$  such that  $({\bf A} - \lambda {\bf I}){\bf x} = 0$ . The linear homogeneous equation system can have a non-zero solution only if  $\det({\bf A} - \lambda {\bf I}) = 0$ . The eigenvalues are obtained as solutions to this characteristic equation. The characteristic equation for the eigenvalues of  $\Omega^2$  is

$$\det(\mathbf{\Omega}^2 - \lambda \mathbf{I}) = \underline{\hspace{1cm}} = 0.$$

The two solutions for the eigenvalues are  $((a-\lambda)^2 - b^2 = 0 \iff \lambda = a \pm b)$ 

$$\lambda_1 = \underline{\hspace{1cm}}$$
 and  $\lambda_2 = \underline{\hspace{1cm}}$ .

The corresponding eigenvectors are obtained as solutions to  $(\mathbf{A} - \lambda \mathbf{I})\mathbf{x} = 0$ . The eigenvectors are not unique and it is enough to find some of them. However, the eigenvectors should be linearly independent so that, e.g., the zero vector is not a valid choice.

$$\lambda_1$$
:  $\begin{bmatrix} x_1 \\ x_2 \end{bmatrix} = 0 \implies \mathbf{x}_1 = \begin{cases} x_1 \\ x_2 \end{bmatrix} = \begin{cases} \underline{\qquad} \end{cases}$ 

$$\lambda_2: \qquad \left[ \begin{array}{ccc} x_1 \\ x_2 \end{array} \right] = 0 \quad \Rightarrow \quad \mathbf{x}_2 = \left\{ \begin{array}{c} x_1 \\ x_2 \end{array} \right\} = \left\{ \begin{array}{ccc} \end{array} \right\}.$$

The representation of the matrix in terms of its eigenvalues and eigenvectors  $\Omega^2 = X\lambda X^{-1}$  implies that  $\Omega = X\sqrt{\lambda}X^{-1}$ . As taking a square root of the diagonal matrix means just taking the square roots of the diagonal terms, the angular speeds of the free vibrations

$$(\omega_1, \mathbf{x}_1) = (\sqrt{\lambda_1}, \mathbf{x}_1) = (\underline{\hspace{1cm}}, \left\{\underline{\hspace{1cm}}\right\}) \quad \text{and} \quad (\omega_2, \mathbf{x}_2) = (\sqrt{\lambda_2}, \mathbf{x}_2) = (\underline{\hspace{1cm}}, \left\{\underline{\hspace{1cm}}\right\}). \quad \longleftarrow$$