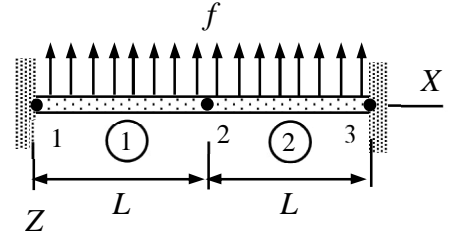


Name _____ Student number _____

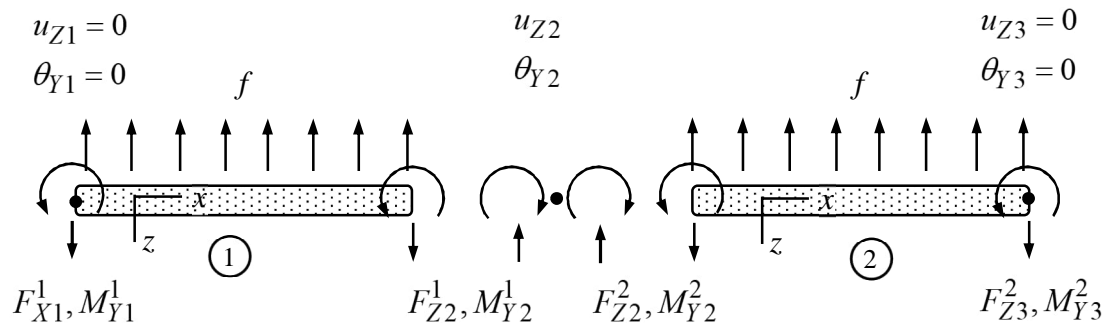
Assignment 5

External load f acting on the beam shown is constant. Determine displacement u_{Z2} and rotation θ_{Y2} of the mid-point (node 2). Young's modulus of the material and the second moments of area are E and I , respectively. Use two beam elements of equal length.



Solution

Only the displacement in the Z – direction and rotation in the Y – direction matter in the planar beam bending problem. From the figure, the non-zero displacement/rotation components are u_{Z2} and θ_{Y2} . Free body diagrams of the two bending beam elements and node 2 are (nodes 1 and 3 are constrained)



Element contributions of the two xz – plane bending beams (formulae collection) and the equilibrium equations of node 2 are (notice that the distributed force in the element contribution is the transverse component in the material system associated with beam)

$$\text{Beam 1: } \begin{Bmatrix} F_{Z1}^1 \\ M_{Y1}^1 \\ F_{Z2}^1 \\ M_{Y2}^1 \end{Bmatrix} = \frac{EI}{L^3} \begin{bmatrix} 12 & -6L & -12 & -6L \\ -6L & 4L^2 & 6L & 2L^2 \\ -12 & 6L & 12 & 6L \\ -6L & 2L^2 & 6L & 4L^2 \end{bmatrix} \begin{Bmatrix} 0 \\ 0 \\ u_{Z2} \\ \theta_{Y2} \end{Bmatrix} + \frac{fL}{12} \begin{Bmatrix} 6 \\ -L \\ 6 \\ L \end{Bmatrix}, \quad (f_z = -f)$$

$$\text{Beam 2: } \begin{Bmatrix} F_{Z2}^2 \\ M_{Y2}^2 \\ F_{Z3}^2 \\ M_{Y3}^2 \end{Bmatrix} = \frac{EI}{L^3} \begin{bmatrix} 12 & -6L & -12 & -6L \\ -6L & 4L^2 & 6L & 2L^2 \\ -12 & 6L & 12 & 6L \\ -6L & 2L^2 & 6L & 4L^2 \end{bmatrix} \begin{Bmatrix} u_{Z2} \\ \theta_{Y2} \\ 0 \\ 0 \end{Bmatrix} + \frac{fL}{12} \begin{Bmatrix} 6 \\ -L \\ 6 \\ L \end{Bmatrix}, \quad (f_z = -f)$$

$$\text{Node 2: } -F_{Z2}^1 - F_{Z2}^2 = 0 \quad \text{and} \quad -M_{Y2}^1 - M_{Y2}^2 = 0.$$

Elimination of the internal forces from the two equilibrium equations of node 2 using the element contributions gives the forms

$$\begin{aligned} \text{Node 2: } & -\left[\frac{EI}{L^3}(12u_{Z2} + 6L\theta_{Y2}) + 6\frac{fL}{12}\right] - \left[\frac{EI}{L^3}(12u_{Z2} - 6L\theta_{Y2}) + 6\frac{fL}{12}\right] = 0 \quad \text{and} \\ & -\left[\frac{EI}{L^3}(6Lu_{Z2} + 4L^2\theta_{Y2}) + L\frac{fL}{12}\right] - \left[\frac{EI}{L^3}(-6Lu_{Z2} + 4L^2\theta_{Y2}) - L\frac{fL}{12}\right] = 0. \end{aligned}$$

Matrix representation of the two equilibrium equations, containing u_{Z2} and θ_{Y2} as the unknowns, is

$$\frac{EI}{L^3} \begin{bmatrix} 24 & 0 \\ 0 & 8L^2 \end{bmatrix} \begin{Bmatrix} u_{Z2} \\ \theta_{Y2} \end{Bmatrix} + fL \begin{Bmatrix} 1 \\ 0 \end{Bmatrix} = 0 \quad \Leftrightarrow$$

$$\begin{Bmatrix} u_{Z2} \\ \theta_{Y2} \end{Bmatrix} = -\frac{fL^4}{EI} \begin{bmatrix} 24 & 0 \\ 0 & 8L^2 \end{bmatrix}^{-1} \begin{Bmatrix} 1 \\ 0 \end{Bmatrix} = -\frac{fL^4}{EI} \begin{bmatrix} 1/24 & 0 \\ 0 & 1/(8L^2) \end{bmatrix} \begin{Bmatrix} 1 \\ 0 \end{Bmatrix} = -\frac{1}{24} \frac{fL^4}{EI} \begin{Bmatrix} 1 \\ 0 \end{Bmatrix} \quad \Leftrightarrow$$

$$u_{Z2} = -\frac{1}{24} \frac{fL^4}{EI} \quad \text{and} \quad \theta_{Y2} = 0. \quad \leftarrow$$