MEC-E1050 Finite Element Method in Solids; Formulae

LINEAR ELASTICITY

Strain-stress:
$$\begin{cases} \varepsilon_{xx} \\ \varepsilon_{yy} \\ \varepsilon_{zz} \end{cases} = \frac{1}{E} \begin{bmatrix} 1 & -\nu & -\nu \\ -\nu & 1 & -\nu \\ -\nu & -\nu & 1 \end{bmatrix} \begin{cases} \sigma_{xx} \\ \sigma_{yy} \\ \sigma_{zz} \end{cases}, \begin{cases} \gamma_{xy} \\ \gamma_{yz} \\ \gamma_{zx} \end{cases} = \frac{1}{G} \begin{cases} \sigma_{xy} \\ \sigma_{yz} \\ \sigma_{zx} \end{cases}, G = \frac{E}{2(1+\nu)} \text{ or }$$

$$\begin{cases}
\sigma_{xx} \\
\sigma_{yy} \\
\sigma_{zz}
\end{cases} = \frac{E}{(1+\nu)(1-2\nu)} \begin{bmatrix}
1-\nu & \nu & \nu \\
\nu & 1-\nu & \nu \\
\nu & \nu & 1-\nu
\end{bmatrix} \begin{bmatrix}
\varepsilon_{xx} \\
\varepsilon_{yy} \\
\varepsilon_{zz}
\end{bmatrix} = [E] \begin{cases}
\varepsilon_{xx} \\
\varepsilon_{yy} \\
\varepsilon_{zz}
\end{cases}, \begin{cases}
\sigma_{xy} \\
\sigma_{yz} \\
\sigma_{zx}
\end{cases} = G \begin{cases}
\gamma_{xy} \\
\gamma_{yz} \\
\gamma_{zx}
\end{cases}$$

$$[E]_{\sigma} = \frac{E}{1 - v^2} \begin{bmatrix} 1 & v & 0 \\ v & 1 & 0 \\ 0 & 0 & (1 - v)/2 \end{bmatrix}, \ [E]_{\varepsilon} = \frac{E}{(1 + v)(1 - 2v)} \begin{bmatrix} 1 - v & v & 0 \\ v & 1 - v & 0 \\ 0 & 0 & (1 - 2v)/2 \end{bmatrix}$$

Strain-displacement:
$$\begin{cases} \varepsilon_{xx} \\ \varepsilon_{yy} \\ \varepsilon_{zz} \end{cases} = \begin{cases} \partial u_x / \partial x \\ \partial u_y / \partial y \\ \partial u_z / \partial z \end{cases}, \begin{cases} \gamma_{xy} \\ \gamma_{yz} \\ \gamma_{zx} \end{cases} = \begin{cases} \partial u_x / \partial y + \partial u_y / \partial x \\ \partial u_y / \partial z + \partial u_z / \partial y \\ \partial u_z / \partial x + \partial u_x / \partial z \end{cases}$$

ELEMENT CONTRIBUTION (constant load)

Bar (axial):
$$\begin{cases} F_{x1} \\ F_{x2} \end{cases} = \frac{EA}{h} \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix} \begin{cases} u_{x1} \\ u_{x2} \end{cases} - \frac{f_x h}{2} \begin{cases} 1 \\ 1 \end{cases}$$

$$\begin{Bmatrix} \mathbf{R}_1 \\ \mathbf{R}_2 \end{Bmatrix} = \frac{EA}{h} \begin{bmatrix} \mathbf{i} \mathbf{i}^{\mathrm{T}} & -\mathbf{i} \mathbf{i}^{\mathrm{T}} \\ -\mathbf{i} \mathbf{i}^{\mathrm{T}} & \mathbf{i} \mathbf{i}^{\mathrm{T}} \end{bmatrix} \begin{Bmatrix} \mathbf{a}_1 \\ \mathbf{a}_2 \end{Bmatrix} - \frac{f_x h}{2} \begin{Bmatrix} \mathbf{i} \\ \mathbf{i} \end{Bmatrix}, \text{ in which } \mathbf{i} = \frac{1}{h} \begin{Bmatrix} \Delta X \\ \Delta Y \\ \Delta Z \end{Bmatrix}$$

Bar (torsion):
$$\begin{cases} M_{x1} \\ M_{x2} \end{cases} = \frac{GI_{rr}}{h} \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix} \begin{cases} \theta_{x1} \\ \theta_{x2} \end{cases} - \frac{m_x h}{2} \begin{cases} 1 \\ 1 \end{cases}$$

$$\mathbf{Beam} \ (\mathbf{xz}) : \begin{cases} F_{z1} \\ M_{y1} \\ F_{z2} \\ M_{y2} \end{cases} = \frac{EI_{yy}}{h^3} \begin{bmatrix} 12 & -6h & -12 & -6h \\ -6h & 4h^2 & 6h & 2h^2 \\ -12 & 6h & 12 & 6h \\ -6h & 2h^2 & 6h & 4h^2 \end{bmatrix} \begin{bmatrix} u_{z1} \\ \theta_{y1} \\ u_{z2} \\ \theta_{y2} \end{bmatrix} - \frac{f_z h}{12} \begin{bmatrix} 6 \\ -h \\ 6 \\ h \end{bmatrix}$$

$$\textbf{Point loads:} \ \begin{cases} F_{X1} \\ F_{Y1} \\ F_{Z1} \end{cases} = \begin{bmatrix} 0 & 0 & 0 \\ 0 & 0 & 0 \\ 0 & 0 & 0 \end{bmatrix} \begin{bmatrix} u_{X1} \\ u_{Y1} \\ u_{Z1} \end{bmatrix} - \begin{bmatrix} F_X \\ F_Y \\ F_Z \end{bmatrix}, \ \begin{cases} M_{X1} \\ M_{Y1} \\ M_{Z1} \end{bmatrix} = \begin{bmatrix} 0 & 0 & 0 \\ 0 & 0 & 0 \\ 0 & 0 & 0 \end{bmatrix} \begin{bmatrix} \theta_{X1} \\ \theta_{Y1} \\ \theta_{Z1} \end{bmatrix} - \begin{bmatrix} M_X \\ M_Y \\ M_Z \end{bmatrix}$$

PRINCIPLE OF VIRTUAL WORK

$$\delta W = \delta W^{\text{ext}} + \delta W^{\text{int}}, \ \delta W = \sum_{e \in E} \delta W^e = 0 \ \forall \delta \mathbf{a}, \ \delta W = \int_{\Omega} \delta w d\Omega$$

Bar:
$$\delta w^{\text{int}} = -\frac{d\delta u}{dx} EA \frac{du}{dx}$$
, $\delta w^{\text{ext}} = \delta u f_x$

Torsion:
$$\delta w^{\text{int}} = -\frac{d\delta\phi}{dx}GI_{rr}\frac{d\phi}{dx}$$
, $\delta w^{\text{ext}} = \delta\phi m_x$

Beam bending (
$$xz$$
-plane): $\delta w^{\text{int}} = -\frac{d^2 \delta w}{dx^2} EI_{yy} \frac{d^2 w}{dx^2}$, $\delta w^{\text{ext}} = \delta w f_z$

Beam bending (xy-plane):
$$\delta w^{\text{int}} = -\frac{d^2 \delta v}{dx^2} E I_{zz} \frac{d^2 v}{dx^2}$$
, $\delta w^{\text{ext}} = \delta v f_y$

Beam (Bernoulli):

$$\delta w^{\rm int} = - \begin{cases} d\delta u/dx \\ d^2\delta v/dx^2 \\ d^2\delta w/dx^2 \end{cases}^{\rm T} E \begin{bmatrix} A & -S_z & -S_y \\ -S_z & I_{zz} & I_{zy} \\ -S_y & I_{yz} & I_{yy} \end{bmatrix} \begin{cases} du/dx \\ d^2v/dx^2 \\ d^2w/dx^2 \end{cases} - \frac{d\delta\phi}{dx} GI_{rr} \frac{d\phi}{dx},$$

$$\delta w^{\text{ext}} = \begin{cases} \delta u \\ \delta v \\ \delta w \end{cases}^{\text{T}} \begin{cases} f_x \\ f_y \\ f_z \end{cases} + \begin{cases} \delta \phi \\ -d \delta w / dx \end{cases}^{\text{T}} \begin{cases} -S_y f_y + S_z f_z \\ S_y f_x \\ -S_z f_z \end{cases}$$

Thin slab (plane-stress):

$$\delta w^{\text{int}} = - \begin{cases} \frac{\partial \delta u / \partial x}{\partial \delta v / \partial y} \\ \frac{\partial \delta u / \partial y + \partial \delta v / \partial x}{\partial v / \partial x} \end{cases}^{\text{T}} t[E]_{\sigma} \begin{cases} \frac{\partial u / \partial x}{\partial v / \partial y} \\ \frac{\partial v / \partial y}{\partial u / \partial y + \partial v / \partial x} \end{cases}, \ \delta w^{\text{ext}} = \begin{cases} \frac{\delta u}{\delta v} \end{cases}^{\text{T}} \begin{cases} f_x \\ f_y \end{cases}$$

Thin slab (plane-strain):

$$\delta w^{\text{int}} = - \begin{cases} \frac{\partial \delta u / \partial x}{\partial \delta v / \partial y} \\ \frac{\partial \delta u / \partial y + \partial \delta v / \partial x}{\partial v / \partial x} \end{cases}^{\text{T}} t[E]_{\varepsilon} \begin{cases} \frac{\partial u / \partial x}{\partial v / \partial y} \\ \frac{\partial u / \partial y + \partial v / \partial x}{\partial v / \partial x} \end{cases}, \ \delta w^{\text{ext}} = \begin{cases} \frac{\delta u}{\delta v} \end{cases}^{\text{T}} \begin{cases} f_x \\ f_y \end{cases}$$

Kirchhoff plate:

$$\delta w^{\text{int}} = - \begin{cases} \partial^2 \delta w / \partial x^2 \\ \partial^2 \delta w / \partial y^2 \\ 2 \partial^2 \delta w / (\partial x \partial y) \end{cases}^{\text{T}} \frac{t^3}{12} [E]_{\sigma} \begin{cases} \partial^2 w / \partial x^2 \\ \partial^2 w / \partial y^2 \\ 2 \partial^2 w / (\partial x \partial y) \end{cases}, \ \delta w^{\text{ext}} = \delta w f_z$$

Reissner-Mindlin plate:

$$\delta w^{\text{int}} = - \begin{cases} -\partial \delta \theta / \partial x \\ \partial \delta \phi / \partial y \\ \partial \delta \phi / \partial x - \partial \delta \theta / \partial y \end{cases} T \frac{t^3}{12} [E]_{\sigma} \begin{cases} -\partial \theta / \partial x \\ \partial \phi / \partial y \\ \partial \phi / \partial x - \partial \theta / \partial y \end{cases} - \begin{cases} \partial \delta w / \partial y - \delta \phi \\ \partial \delta w / \partial x + \delta \theta \end{cases} T t G \begin{cases} \partial w / \partial y - \phi \\ \partial w / \partial x + \theta \end{cases},$$

$$\delta w^{\text{ext}} = \delta w f_z$$

Body:
$$\delta w^{\text{int}} = -\begin{cases} \delta \varepsilon_{xx} \\ \delta \varepsilon_{yy} \\ \delta \varepsilon_{zz} \end{cases}^{\text{T}} \begin{cases} \sigma_{xx} \\ \sigma_{yy} \\ \sigma_{zz} \end{cases} - \begin{cases} \delta \gamma_{xy} \\ \delta \gamma_{yz} \\ \delta \gamma_{zx} \end{cases}^{\text{T}} \begin{cases} \sigma_{xy} \\ \sigma_{yz} \\ \sigma_{zx} \end{cases}, \ \delta w^{\text{ext}} = \begin{cases} \delta u \\ \delta v \\ \delta w \end{cases}^{\text{T}} \begin{cases} f_x \\ f_y \\ f_z \end{cases} \text{ or }$$

$$\delta w^{\text{int}} = - \begin{cases} \partial \delta u / \partial x \\ \partial \delta v / \partial y \\ \partial \delta w / \partial z \end{cases}^{\text{T}} \begin{bmatrix} E \end{bmatrix} \begin{cases} \partial u / \partial x \\ \partial v / \partial y \\ \partial w / \partial z \end{cases} - \begin{cases} \partial \delta u / \partial y + \partial \delta v / \partial x \\ \partial \delta v / \partial z + \partial \delta w / \partial y \end{cases}^{\text{T}} G \begin{cases} \partial u / \partial y + \partial v / \partial x \\ \partial v / \partial z + \partial w / \partial y \end{cases}$$

APPROXIMATIONS (some) $u = \mathbf{N}^{\mathrm{T}} \mathbf{a}$, $\xi = \frac{x}{h}$

Quadratic line:
$$\mathbf{N} = \begin{cases} N_1 \\ N_2 \\ N_3 \end{cases} = \begin{cases} 1 - 3\xi + 2\xi^2 \\ 4\xi(1 - \xi) \\ \xi(2\xi - 1) \end{cases}$$
, $\mathbf{a} = \begin{cases} u_{x1} \\ u_{x2} \\ u_{x3} \end{cases}$ (bar)

Cubic line:
$$\mathbf{N} = \begin{cases} N_{10} \\ N_{11} \\ N_{20} \\ N_{21} \end{cases} = \begin{cases} \frac{(1-\xi)^2(1+2\xi)}{h(1-\xi)^2\xi} \\ \frac{h(1-\xi)^2\xi}{(3-2\xi)\xi^2} \\ h\xi^2(\xi-1) \end{cases}$$
, $\mathbf{a} = \begin{cases} u_{10} \\ u_{11} \\ u_{20} \\ u_{21} \end{cases} (= \begin{cases} u_{z1} \\ -\theta_{y1} \\ u_{z2} \\ -\theta_{y2} \end{cases})$ (beam bending)

Linear:
$$\mathbf{N} = \begin{bmatrix} 1 & 1 \\ x_1 & x_2 \end{bmatrix}^{-1} \begin{bmatrix} 1 \\ x \end{bmatrix}, \quad \mathbf{N} = \begin{bmatrix} 1 & 1 & 1 \\ x_1 & x_2 & x_3 \\ y_1 & y_2 & y_3 \end{bmatrix}^{-1} \begin{bmatrix} 1 \\ x \\ y \end{bmatrix}, \quad \mathbf{N} = \begin{bmatrix} 1 & 1 & 1 & 1 \\ x_1 & x_2 & x_3 & x_4 \\ y_1 & y_2 & y_3 & y_4 \\ z_1 & z_2 & z_3 & z_4 \end{bmatrix}^{-1} \begin{bmatrix} 1 \\ x \\ y \\ z \end{bmatrix}$$

VIRTUAL WORK EXPRESSIONS

Force:
$$\delta W^{\text{ext}} = \begin{cases} \delta u_{Xi} \\ \delta u_{Yi} \\ \delta u_{Zi} \end{cases}^{\text{T}} \begin{cases} F_{Xi} \\ F_{Yi} \\ F_{Zi} \end{cases} + \begin{cases} \delta \theta_{Xi} \\ \delta \theta_{Yi} \\ \delta \theta_{Zi} \end{cases}^{\text{T}} \begin{cases} M_{Xi} \\ M_{Yi} \\ M_{Zi} \end{cases}$$

Bar:
$$\delta W^{\text{int}} = -\begin{cases} \delta u_{x1} \\ \delta u_{x2} \end{cases}^{\text{T}} \frac{EA}{h} \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix} \begin{cases} u_{x1} \\ u_{x2} \end{cases}, \ \delta W^{\text{ext}} = \begin{cases} \delta u_{x1} \\ \delta u_{x2} \end{cases}^{\text{T}} \frac{f_x h}{2} \begin{bmatrix} 1 \\ 1 \end{bmatrix}$$

Torsion:
$$\delta W^{\text{int}} = -\begin{cases} \delta \theta_{x1} \\ \delta \theta_{x2} \end{cases}^{\text{T}} \frac{GI_{rr}}{h} \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix} \begin{cases} \theta_{x1} \\ \theta_{x2} \end{cases}, \ \delta W^{\text{ext}} = \begin{cases} \delta \theta_{x1} \\ \delta \theta_{x2} \end{cases}^{\text{T}} \frac{m_x h}{2} \begin{cases} 1 \\ 1 \end{cases})$$

Beam bending (xz-plane):

$$\delta W^{\text{int}} = - \begin{cases} \delta u_{z1} \\ \delta \theta_{y1} \\ \delta u_{z2} \\ \delta \theta_{y2} \end{cases}^{\text{T}} \frac{EI_{yy}}{h^{3}} \begin{bmatrix} 12 & -6h & -12 & -6h \\ -6h & 4h^{2} & 6h & 2h^{2} \\ -12 & 6h & 12 & 6h \\ -6h & 2h^{2} & 6h & 4h^{2} \end{bmatrix} \begin{bmatrix} u_{z1} \\ \theta_{y1} \\ u_{z2} \\ \theta_{y2} \end{cases}, \ \delta W^{\text{ext}} = \begin{cases} \delta u_{z1} \\ \delta \theta_{y1} \\ \delta u_{z2} \\ \delta \theta_{y2} \end{cases}^{\text{T}} \frac{f_{z}h}{12} \begin{bmatrix} 6 \\ -h \\ 6 \\ h \end{cases}$$

Beam bending (xy-plane):

$$\delta W^{\text{int}} = - \begin{cases} \delta u_{y1} \\ \delta \theta_{z1} \\ \delta u_{y2} \\ \delta \theta_{z2} \end{cases}^{\text{T}} \begin{bmatrix} 12 & 6h & -12 & 6h \\ 6h & 4h^2 & -6h & 2h^2 \\ -12 & -6h & 12 & -6h \\ 6h & 2h^2 & -6h & 4h^2 \end{bmatrix} \begin{bmatrix} u_{y1} \\ \theta_{z1} \\ u_{y2} \\ \theta_{z2} \end{bmatrix}, \ \delta W^{\text{ext}} = \begin{cases} \delta u_{y1} \\ \delta \theta_{z1} \\ \delta u_{y2} \\ \delta \theta_{z2} \end{cases}^{\text{T}} \frac{f_y h}{12} \begin{cases} 6 \\ h \\ 6 \\ -h \end{cases})$$

CONSTRAINTS

Frictionless contact: $\vec{n} \cdot \vec{u}_A = 0$

Joint: $\vec{u}_{\rm B} = \vec{u}_{\rm A}$

Rigid body (link): $\vec{u}_B = \vec{u}_A + \vec{\theta}_A \times \vec{\rho}_{AB}$, $\vec{\theta}_B = \vec{\theta}_A$.