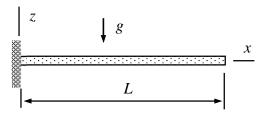
## **Assignment 1**

Consider the beam loaded by its own weight as shown in the figure. Thickness, width, and length of the beam are t, b, and L, respectively. Density  $\rho$ , Young's modulus E, and Poisson's ratio  $\nu$  are constants. Find the unknown parameter  $a_0$  of the assumed transverse displacement  $w = a_0 x^2$ . The origin of the material coordinate system is placed at the symmetry axes of the rectangular cross section.



## **Solution template**

Virtual work density expressions of the beam bending mode are

$$\delta w_{\Omega}^{\text{int}} = -\frac{d^2 \delta w}{dx^2} E I_{yy} \frac{d^2 w}{dx^2} \text{ and } \delta w_{\Omega}^{\text{ext}} = \delta w f_z,$$

in which  $f_z$  is the z-component of the external force per unit length, E is the Young's modulus of the material, and  $I_{yy}$  the second moment of area with respect to the area centroid.

With the displacement assumption  $w = a_0 x^2$  and the expression of  $I_{yy}$ , virtual work densities simplify to

$$\delta w_{\Omega}^{\text{int}} = -\frac{d^2 \delta w}{dx^2} E I_{yy} \frac{d^2 w}{dx^2} = -\delta a_0 \frac{Ebt^3}{3} a_0,$$

$$\delta w_{\Omega}^{\rm ext} = \delta w f_z = -\delta a_0 x^2 g \rho t b$$
.

Integration over the length of the beam gives the virtual work expressions

$$\delta W^{\rm int} = \int_0^L \delta w_{\Omega}^{\rm int} dx = -\delta a_0 \frac{Ebt^3 L}{3} a_0,$$

$$\delta W^{\rm ext} = \int_0^L \delta w_{\Omega}^{\rm ext} dx = -\delta a_0 \frac{1}{3} L^3 g \, \rho t b \,.$$

Finally, principle of virtual work with  $\delta W = \delta W^{\rm int} + \delta W^{\rm ext}$  and the fundamental lemma of variation calculus imply the solution

$$\delta W = -\delta a_0 \left( \frac{Ebt^3 L}{3} a_0 + \frac{1}{3} L^3 g \, \rho tb \right) = 0 \quad \forall \, \delta a_0 \quad \Leftrightarrow \quad$$

$$\frac{Ebt^3L}{3}a_0 + \frac{1}{3}L^3g\,\rho tb = 0 \quad \Leftrightarrow \quad$$

$$a_0 = -(\frac{L}{t})^2 \frac{g\rho}{E}.$$