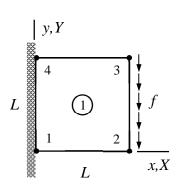
Assignment 4

A thin slab is loaded by distributed force on its outer edge as shown in the figure. Determine the vertical displacement of the outer edge 2-3 by using a bi-linear interpolation to the nodal values. Edge 1-4 is welded to a rigid wall so that the displacements vanish. Thickness of the slab t, Young's modulus E, and Poisson's ratio ν are constants. Assume plane stress conditions. Simplify the setting with conditions $u_{Y3} = u_{Y2}$ and $u_{X3} = u_{X2} = 0$.



Solution template

Under the plane stress conditions, the virtual work densities (virtual works per unit area) of the thin slab model are given by

$$\delta w_{\Omega}^{\text{int}} = -\left\{ \begin{array}{c} \partial \delta u / \partial x \\ \partial \delta v / \partial y \\ \partial \delta u / \partial y + \partial \delta v / \partial x \end{array} \right\}^{\text{T}} t[E]_{\sigma} \left\{ \begin{array}{c} \partial u / \partial x \\ \partial v / \partial y \\ \partial u / \partial y + \partial v / \partial x \end{array} \right\} \text{ and } \delta w_{\Omega}^{\text{ext}} = \left\{ \begin{array}{c} \delta u \\ \delta v \end{array} \right\}^{\text{T}} \left\{ \begin{array}{c} f_x \\ f_y \end{array} \right\} \text{ where }$$

$$[E]_{\sigma} = \frac{E}{1 - v^2} \begin{bmatrix} 1 & v & 0 \\ v & 1 & 0 \\ 0 & 0 & (1 - v)/2 \end{bmatrix}.$$

Expressions take into account the internal forces (stress) and the external area forces acting on the element domain. The external forces t_x and t_y (tractions per unit length in this case) acting on the element edges can be taken into account by a separate force element with the density expression (per unit length)

$$\delta w_{\partial\Omega}^{\text{ext}} = \begin{bmatrix} \delta u \\ \delta v \end{bmatrix}^{\text{T}} \begin{bmatrix} t_x \\ t_y \end{bmatrix}.$$

The approximation on the boundary is just the restriction of the element approximation to the boundary (corresponds to a linear two-node element).

Only the shape functions associated with nodes 2 and 3 are needed as the other nodes are fixed (displacement vanishes). By deducing the expression, i.e., combining the linear shape functions in the x-directions and y-directions

$$N_2 =$$
 _____ and $N_3 =$ _____.

In terms of the vertical displacement component u_{Y2} of node 2, approximations to the displacement components and their derivatives are

$$u(x, y) = \mathbf{N}^{\mathrm{T}} \mathbf{a} = \underline{\qquad} \Rightarrow \frac{\partial u}{\partial x} = \underline{\qquad} \text{and } \frac{\partial u}{\partial y} = \underline{\qquad},$$

$$v(x, y) = \mathbf{N}^{\mathrm{T}} \mathbf{a} = \underline{\qquad} \qquad \Rightarrow \frac{\partial v}{\partial x} = \underline{\qquad} \qquad \text{and} \quad \frac{\partial v}{\partial y} = \underline{\qquad}.$$

Virtual work density of the internal forces simplifies to (when the approximations are substituted there)

$$\delta w_{\Omega}^{\rm int} = - \left\{ \begin{array}{c} \\ \\ \end{array} \right\}^{\rm T} \frac{Et}{1-v^2} \begin{bmatrix} 1 & v & 0 \\ v & 1 & 0 \\ 0 & 0 & (1-v)/2 \end{bmatrix} \left\{ \begin{array}{c} \\ \\ \end{array} \right\} = \underline{\hspace{2cm}}.$$

Virtual work density is constant in this case. Integration over the element gives the virtual work expression of internal forces

$$\delta W^{\rm int} = \int_0^L \int_0^L \delta w_{\Omega}^{\rm int} dx dy = \underline{\hspace{1cm}}.$$

Virtual work expression of external distributed force components $t_x = 0$ and $t_y = -f$ is obtained as an integral over the edge defined by x = L. The restriction of approximation to x = L is given by

$$u(L, y) =$$
_____ and $v(L, y) =$ _____

so the virtual work density expression simplifies to

giving the virtual work expression

$$\delta W^{\text{ext}} = \int_0^L w_{\partial\Omega}^{\text{ext}} dy = \underline{\qquad}.$$

Virtual work expression is the sum of internal and external parts

$$\delta W = \delta W^{\text{int}} + \delta W^{\text{ext}} =$$

Principle of virtual work $\delta W = 0 \ \forall \delta \mathbf{a}$ and the fundamental lemma of variation calculus in the form $\delta \mathbf{a}^T \mathbf{R} = 0 \ \forall \delta \mathbf{a} \Leftrightarrow \mathbf{R} = 0 \ \text{give}$

$$\underline{\hspace{1cm}} = 0 \Leftrightarrow u_{Y2} = \underline{\hspace{1cm}}.$$