# Foundations on Continuum Mechanics - Week 6 - Constitutive Equations - Fluids

Athanasios A. Markou

PhD, University Lecturer Aalto University School of Engineering Department of Civil Engineering

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# **Fluids**

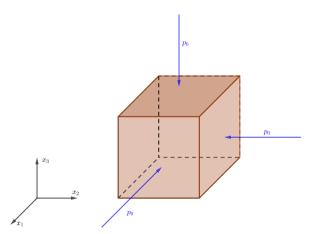
### What is fluid?

#### Fluids can be classified in:

- ► Ideal (inviscid) fluids:
  - Called also perfect fluid
  - Water is an example of inviscid fluid
  - Resists only compressive stresses (pressure)
  - No resistance in fluid moves
- ► Real (viscous) fluids:
  - ► Has viscous (friction) behavior
  - ► Honey is an example of viscous fluid
  - Resistance in fluid moves

### Pascal's Law

For a fluid at rest, the pressure acts on all directions.



### Consequences of Pascal's Law

#### Consequences:

- ► No shear stresses at rest
- ▶ Only normal stresses due to pressure

The stress is isotropic at rest and of the form:

$$\underline{\underline{\sigma}} = -p_0 \underline{\underline{1}}$$

$$\sigma_{ij} = -p_0 \delta_{ij} \quad i, j \in \{1, 2, 3\}$$

 $p_0$  denotes the hydrostatic pressure.

### Pressure Concepts

#### Consequences:

- ightharpoonup Hydrostatic pressure  $p_0$ : normal compressive stress on a fluid.
- ightharpoonup Mean pressure  $\overline{p}$ :

$$\overline{p} = -\sigma_m = -\frac{1}{3} \operatorname{Tr}(\underline{\underline{\sigma}})$$

 $Tr(\underline{\boldsymbol{\sigma}})$  is an invariant and therefore  $\sigma_m$  and  $\overline{p}$  are also.

▶ Thermodynamic pressure *p*: used in constitutive equations. It is related to density and temperature through the kinetic equation of state:

$$F(\rho, p, \theta) = 0$$

For a fluid at rest:  $p_0 = \overline{p} = p$ 

### Pressure Concepts

#### Consequences:

▶ Barotropic fluid is the fluid in which the pressure depends only on the density, [1]:

$$F(\rho, p) = 0 \Rightarrow p = f(\rho)$$

Incompressible fluid, when the density is constant (special case of barometric):

$$F(\rho, p, \theta) \equiv F(\rho) = \rho - k = 0 \Rightarrow p = f(\rho) \Rightarrow \rho = k = const.$$

# **Constitutive Equations**

### Governing Equations: Thermo-mechanical problem

$\dot{\rho} + \rho \vec{\nabla} \cdot \vec{v} = 0$	Conservation of Mass	1 eqn.	PDE
$ \rho + \rho \mathbf{v} \cdot \mathbf{v} \equiv 0 $	Continuity Equation		
$egin{equation} ec{m{ abla}} \cdot oldsymbol{ar{\sigma}} +  ho ec{m{b}} =  ho \dot{ec{m{v}}} \end{aligned}$	Linear Momentum Balance	3 eqns.	PDE
$oldsymbol{v}\cdot\underline{\underline{oldsymbol{arrho}}}+ hooldsymbol{oldsymbol{v}}= hooldsymbol{v}$	Cauchy's Equation of motion		
$\overline{\qquad \qquad }$	Angular Momentum Balance	3 eqns.	ALG
$\underline{\underline{\sigma}} = \underline{\underline{\sigma}}^T$	Symmetry of Cauchy stress tensor		
$\vec{z} = \vec{z} \cdot \vec{d} + \vec{z} \cdot \vec{z}$	Energy Balance	1 eqn.	PDE
$ \rho \dot{u} = \underline{\underline{\sigma}} : \underline{\underline{d}} + \rho \ r - \vec{\nabla} \cdot \vec{q} $	First Law of Thermodynamics		
$-\rho(\dot{u}-\theta\dot{s})+\underline{\boldsymbol{\sigma}}:\underline{\boldsymbol{d}}\geq 0$	Second Law of Thermodynamics		
$-rac{1}{ ho heta^2}ec{m{q}}\cdotec{m{ abla}} heta\geq 0$	Clausius-Plank Inequality	2 restrictions	PDE
$-rac{}{ ho heta^2}oldsymbol{q}\cdotoldsymbol{f V} heta\geq 0$	Heat Flow Inequality		
0.000			

8 PDE and 2 restrictions

19 unknown scalars:  $\rho, \vec{\pmb{v}}, u, \underline{\underline{\pmb{\sigma}}}, \vec{\pmb{q}}, \theta, s$ 

# Constitutive Equations: Thermo-mechanical problem

### The Constitutive Equations:

$\mathbf{r} = \mathbf{r}(\vec{\mathbf{z}}, \theta, \dot{c})$		Thermo-Mechanical	6 egns.	
$\underline{\underline{\boldsymbol{\sigma}}} = \underline{\underline{\boldsymbol{\sigma}}}(\vec{\boldsymbol{v}}, \theta, \zeta)$	Constitutive Equations			
$s = s(\vec{\boldsymbol{v}}, \theta, \zeta)$	Entropy	1 eqn.		
	Constitutive Equation			
$ec{m{q}} = ec{m{q}}(ec{m{v}},  heta) = -k ec{m{\nabla}}  heta$	<del>=</del> <del>=</del> <del>=</del> <del>=</del> <del>=</del> <del>=</del> <del>=</del> 0 1 <del>=</del> <del>=</del> <del>=</del> 0	Thermal Constitutive Equation	2 0000	
	Fourier's Law of Conduction	3 eqns.		
$u = v(\rho, \vec{\boldsymbol{v}}, \theta, \zeta)$		Heat State Equation	(1   n) ogns	
	$F_i(\rho, \theta, \zeta);  i \in \{1, 2,, p\}$	Kinetic State Equation	(1+p) eqns.	

### Constitutive Equations

The constitutive equations along with governing equations can used to solve the problem In fluid mechanics can be grouped as:

Thermo-mechanical constitutive equations:

$$\underline{\underline{\sigma}} = -p\underline{\underline{1}} + \underline{\underline{f}}(\underline{\underline{d}}, \rho, \theta)$$

$$\sigma_{ij} = -p\delta_{ij} + f_{ij}(\underline{\underline{d}}, \rho, \theta); \quad i, j \in \{1, 2, 3\}$$

Caloric equation of state

$$u = g(\rho, \theta)$$

Entropy constitutive equation

$$s=s(\underline{\underline{\boldsymbol{d}}},\rho,\theta)$$

► Fourier's Law:

$$\vec{q} = -k\vec{\nabla}\theta$$
$$q_i = -k\frac{\partial\theta}{\partial x_i} \quad i, j \in \{1, 2, 3\}$$

Kinetic equation of state:

$$F(\rho, p, \theta) = 0$$



### Viscous Fluid Models

General form of the thermo-mechanical constitutive equations:

$$\underline{\underline{\sigma}} = -p\underline{\underline{1}} + \underline{\underline{f}}(\underline{\underline{d}}, \rho, \theta)$$

$$\sigma_{ij} = -p\delta_{ij} + f_{ij}(\underline{\underline{d}}, \rho, \theta); \quad i, j \in \{1, 2, 3\}$$

Fluids are classified as:

- ▶ Perfect fluid (no viscosity),  $\underline{\underline{f}}(\underline{\underline{d}}, \rho, \theta) = 0 \Rightarrow \underline{\underline{\sigma}} = -p\underline{\underline{1}}$ .
- ▶ Newtonian fluid (viscous),  $f(\underline{d}, \rho, \theta)$  is a linear function of strain rate.
- ▶ Stokesian fluid (viscous),  $\underline{f}(\underline{\underline{d}}, \rho, \theta)$  is a non-linear function of its arguments, [1].

Note that  $\underline{\underline{d}} = \frac{1}{2} \left( \vec{v} \otimes \vec{\nabla} + \vec{\nabla} \otimes \vec{v} \right)$  is the symmetric part of the velocity gradient tensor  $\underline{\underline{d}}$ .

### Constitutive Equations: Newtonian Fluids

Constitutive equations:

$$\underline{\underline{\sigma}} = -p\underline{\underline{1}} + \underline{\underline{C}} : \underline{\underline{d}}$$

$$\sigma_{ij} = -p\delta_{ij} + C_{ijkl}d_{kl}; \quad i, j \in \{1, 2, 3\}$$

where  $\underline{\underline{\boldsymbol{p}}}$  is a  $4^{th}$  order constant VISCOUS constitutive tensor.

For an isotropic material the viscous constitutive tensor becomes:

$$\underline{\underline{\underline{C}}} = \lambda \underline{\underline{1}} \otimes \underline{\underline{1}} + 2\mu \underline{\underline{I}}$$

$$C_{ijkl} = \lambda \delta_{ij} \delta_{kl} + \mu (\delta_{ik} \delta_{jl} + \delta_{il} \delta_{jk})$$

Finally in the constitutive equation:

$$\underline{\underline{\boldsymbol{\sigma}}} = -p\underline{\underline{1}} + \lambda \operatorname{Tr}(\underline{\underline{\boldsymbol{d}}})\underline{\underline{1}} + 2\mu\underline{\underline{\boldsymbol{d}}}$$

$$\sigma_{ij} = -p\delta_{ij} + \lambda d_{ll}\delta_{ij} + \mu d_{ij}; \quad i, j \in \{1, 2, 3\}$$

Note that  $\lambda$  and  $\mu$  can vary with  $\rho$  and  $\theta$ .



### Dissipative and Recoverable Parts of Stress Tensor

The mechanical energy balance:

$$\underbrace{P_e(t)}_{\text{mechanical power}} = \int \int \int_V \rho \vec{\boldsymbol{b}} \cdot \vec{\boldsymbol{v}} dV + \int \int_{\partial V} \vec{\boldsymbol{t}} \cdot \vec{\boldsymbol{v}} dS = \frac{d}{dt} \underbrace{\int \int \int_V \frac{1}{2} \rho v^2 dV}_{\text{kinetic energy}} + \underbrace{\int \int \int_V \underline{\boldsymbol{\sigma}}}_{\text{stress power}} : \underline{\underline{\boldsymbol{d}}} dV$$

Then:

$$P_e(t) = \frac{d}{dt}K(t) + P_{\sigma}$$

Note that a rigid body will have zero stress power.

The stress power is mechanical in the system, which is not spent in changing the kinetic energy, [1].

# Dissipative and Recoverable Stress

Knowing that:

$$\int\int\int_{V}\underline{\underline{\boldsymbol{\sigma}}}:\underline{\underline{\boldsymbol{d}}}\ dV;\quad\underline{\underline{\boldsymbol{d}}}=\frac{1}{3}\operatorname{Tr}(\underline{\underline{\boldsymbol{d}}})\underline{\underline{1}}+\underline{\underline{\boldsymbol{d}}}_{dev};\quad\underline{\underline{\boldsymbol{\sigma}}}=-\overline{p}\underline{\underline{1}}+\underline{\underline{\boldsymbol{\sigma}}}_{dev};\quad\overline{p}=-\frac{1}{3}\operatorname{Tr}(\underline{\underline{\boldsymbol{\sigma}}})$$

Then:

$$\underline{\underline{\sigma}} : \underline{\underline{d}} = \left( -\overline{p}\underline{\underline{1}} + \underline{\underline{\sigma}}_{dev} \right) : \left( \frac{1}{3} \operatorname{Tr}(\underline{\underline{d}})\underline{\underline{1}} + \underline{\underline{d}}_{dev} \right) =$$

$$= -\frac{1}{3} \overline{p} \operatorname{Tr}(\underline{\underline{d}}) \underline{\underline{1}} : \underline{\underline{1}} + \underline{\underline{\sigma}}_{dev} : \underline{\underline{d}}_{dev} - \overline{p} \underbrace{\underline{\underline{1}} : \underline{\underline{d}}_{dev}}_{= \operatorname{Tr}(\underline{\underline{d}}_{dev}) = 0} + \frac{1}{3} \operatorname{Tr}(\underline{\underline{d}}) \underbrace{\underline{\underline{\sigma}}_{dev} : \underline{\underline{1}}}_{\operatorname{Tr}(\underline{\underline{\sigma}}_{dev}) = 0} = -\overline{p} \operatorname{Tr}(\underline{\underline{d}}) + \underline{\underline{\sigma}}_{dev} : \underline{\underline{d}}_{dev} : \underline{\underline{d}}_{dev}$$

Noting that  $\underline{\underline{\sigma}}_{dev} = 2\mu \underline{\underline{d}}_{dev}$  and  $\overline{p} = p - \kappa Tr(\underline{\underline{\underline{d}}})$ , we have:

$$\underline{\underline{\sigma}}:\underline{\underline{d}} = \underbrace{-p \ Tr(\underline{\underline{d}})}_{\text{Recoverable Power } W_R} + \underbrace{\kappa Tr^2(\underline{\underline{d}}) + 2\mu \underline{\underline{d}}_{dev}:\underline{\underline{d}}_{dev}}_{\text{Dissipative Power } 2W_D}$$

where  $\kappa = \lambda + \frac{2}{3}\mu$  is the bulk viscosity.

### Dissipative and Recoverable Parts of Stress Tensor

In the Cauchy stress tensor the dissipative and recoverable parts can be split as follows:

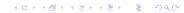
$$\underline{\underline{\sigma}} = \underbrace{-p\underline{\underline{1}}}_{\text{Recoverable Part }\underline{\underline{\sigma}}_R} + \underbrace{\lambda \operatorname{Tr}(\underline{\underline{d}})\underline{\underline{1}} + 2\mu\underline{\underline{d}}}_{\text{Dissipative Part }\underline{\underline{\sigma}}_D}$$

More specifically, the recoverable and the dissipative part can be written:

$$W_R = -p Tr(\underline{\underline{d}}) = -p \underline{\underline{1}} : \underline{\underline{d}} = \underline{\underline{\sigma}}_R : \underline{\underline{d}}$$
$$2 W_D = \kappa Tr^2(\underline{\underline{d}}) + 2\mu \underline{\underline{d}}_{dev} : \underline{\underline{d}}_{dev} = \underline{\underline{\sigma}}_D : \underline{\underline{d}}$$

Note that for incompressible fluid:  $W_R = -p Tr(\underline{\underline{d}}) = 0$ . Due to second principle of thermodynamics:

- ▶ the dissipative part  $2W_D \ge 0$ ,
- ▶ the bulk viscosity  $\kappa = \lambda + \frac{2}{3}\mu \ge 0$
- ▶ the shear viscosity  $\mu \ge 0$ .



# Fluid Mechanics

# **Governing Equations**

$\dot{ ho}+ hoec{m{ abla}}\cdot\dot{m{v}}=0$ Conservation of Mass 1 eqn.	PDE
$ ho +  ho \mathbf{v} \cdot v \equiv 0$ Continuity Equation	
$\vec{\nabla} \cdot \underline{\sigma} + \rho \vec{b} = \dot{\vec{v}}$ Linear Momentum Balance 3 eqns.	PDE
$\mathbf{v}\cdot \mathbf{\underline{\underline{\sigma}}} + \rho \mathbf{v} = \rho \mathbf{v}$ Cauchy's Equation of motion	
Angular Momentum Balance 3 eqns.	ALG
$\underline{\underline{\sigma}} = \underline{\underline{\sigma}}^T$ Symmetry of Cauchy stress tensor	
Energy Balance 1 eqn.	PDE
$ ho\dot{u} = \underline{\boldsymbol{\sigma}} : \underline{\boldsymbol{d}} +  ho \ r - \vec{\boldsymbol{\nabla}} \cdot \vec{\boldsymbol{q}}$ First Law of Thermodynamics	
$-\rho(\dot{u}-\theta\dot{s})+\underline{\underline{\sigma}}:\underline{\underline{d}}\geq 0$ Second Law of Thermodynamics	
$-\frac{1}{a\theta^2} \vec{q} \cdot \vec{\nabla} \theta \geq 0$ Clausius-Plank Inequality 2 restrictions	PDE
$-rac{- ho heta^2}{ ho heta^2} oldsymbol{q}\cdotoldsymbol{\mathbf{v}} oldsymbol{artheta} \geq 0$ Heat Flow Inequality	

8 Eqns and 2 restrictions

### Constitutive Equations: Newtonian Fluids

The 12 missing equations:

$\underline{\underline{\boldsymbol{\sigma}}} = -p\underline{\underline{1}} + \lambda \operatorname{Tr}(\underline{\underline{\boldsymbol{d}}})\underline{\underline{1}} + 2\mu\underline{\underline{\boldsymbol{d}}}$	Thermo-Mechanical	6 cans	
	Constitutive Equations	6 eqns.	
$s = s(\vec{\boldsymbol{v}}, \theta, \zeta)$	Entropy	1 eqn.	
	Constitutive Equation		
$ec{m{q}} = ec{m{q}}(ec{m{v}},  heta) = -k ec{m{\nabla}}  heta$	Thermal Consitutive Equation	2 oans	
	Fourier's Law of Conduction	3 eqns.	
$u = v(\rho, \theta, \zeta)$	Caloric State Equation	2 eqns.	
$F(\rho, p, \theta) = 0$	Kinetic State Equation	z equs.	

In total: 20 Eqns. with 20 unknowns:

$$\rho \to 1, \; \vec{\pmb{v}} \to 3, \; \underline{\underline{\pmb{\sigma}}} \to 9, \; u \to 1, \; \vec{\pmb{q}} \to 3, \; \theta \to 1, \; s \to 1, \; p \to 1$$

### Constitutive Equations: Barotropic Fluids

A barotropic fluid is defined by the kinetic state equation, not depend on temperature  $\theta$ :

$$F(\rho, p) = 0 \Rightarrow \rho = \rho(p)$$

The uncoupled problem becomes:

$\dot{\rho} + \rho \vec{\nabla} \cdot \vec{v} = 0$	Conservation of Mass	1 eqn.
$ ho +  ho \mathbf{v} \cdot \mathbf{v} \equiv 0$	Continuity Equation	
$ec{m{ abla}}\cdot oldsymbol{ec{\sigma}} +  ho ar{m{b}} =  ho \dot{ar{m{v}}}$	Linear Momentum Balance	3 eqns.
	Cauchy's Equation of motion	
$\underline{\underline{\sigma}} = -p\underline{\underline{1}} + \lambda Tr(\underline{\underline{d}})\underline{\underline{1}} + 2\mu\underline{\underline{d}}$	Thermo-Mechanical	6 0000
	Constitutive Equations	6 eqns.
$\rho = \rho(p)$	Kinetic State Equation	1 eqn.
11 and $11$ $11$ $11$ $11$ $11$ $11$ $11$ $11$	-(c) -(1)	

11 scalar unknowns:  $\rho(1), \vec{v}(3), \underline{\sigma}(6), p(1)$ 

# **Hydrostatics**

### Hydrostatic stress state

Uniform velocity:

$$ec{m{v}}(ec{m{x}},t) \equiv ec{m{v}}(ec{m{x}}) \Rightarrow ec{m{\nabla}} ec{m{v}} = ec{m{\nabla}} \otimes ec{m{v}} = ec{m{v}} \otimes ec{m{\nabla}} = \underline{m{0}}$$

$$\underline{m{d}} = \frac{1}{2} \left( ec{m{v}} \otimes ec{m{\nabla}} + ec{m{\nabla}} \otimes ec{m{v}} 
ight) = \underline{m{0}}$$

Therefore, the hydrostatic stress state can be defines as:

$$\underline{\underline{\sigma}} = -p\underline{\underline{1}} + \lambda \underbrace{Tr(\underline{\underline{d}})}_{=0} \underline{\underline{1}} + 2\mu \underbrace{\underline{\underline{d}}}_{=\underline{\underline{0}}} \Rightarrow \underline{\underline{\sigma}} = -p\underline{\underline{1}} \Rightarrow \overline{p} = p$$

▶ Uniform and stationary velocity  $\vec{v}(\vec{x}, t) = \text{constant}$ :

$$ec{m{a}} = rac{dec{m{v}}}{dt} = rac{\partial ec{m{v}}}{\partial t} + ec{m{v}} \cdot ec{m{
abla}} ec{m{v}} = ec{m{0}}$$

There the hydrostatic case can be derived as:

$$\underline{\boldsymbol{\sigma}} = -p_0 \underline{\mathbf{1}} \Rightarrow Tr(\underline{\boldsymbol{\sigma}}) = -3p_0 \Rightarrow \overline{p} = p = p_0$$

Fluid at rest  $\vec{v}(\vec{x},t)=\text{constant}=\vec{0}$ . Hydrostatic (from Greek and means water at rest).

# Hydrostatic Problem

Hydrostatic problem  $\vec{v}(\vec{x},t) = \text{constant}$  is defined as:

$\vec{\mathbf{x}}$	Conservation of Mass	1 eqn.
$\dot{\rho} + \rho \vec{\nabla} \cdot \vec{v} = 0 \Rightarrow \rho(\vec{X}, t) = \rho_0(\vec{X})$	Continuity Equation	
$oxed{ec{ abla}\cdot\underline{oldsymbol{\sigma}}+ hoec{oldsymbol{b}}= ho\dot{ec{oldsymbol{v}}}\Rightarrowec{oldsymbol{ abla}\cdot\underline{oldsymbol{\sigma}}+ hoec{oldsymbol{b}}=ec{oldsymbol{0}}$	Linear Momentum Balance	3 eqns.
	Cauchy's Equation of motion	
$\underline{\underline{\sigma}} = -p\underline{\underline{1}} + \lambda Tr(\underline{\underline{d}})\underline{\underline{1}} + 2\mu \underline{\underline{d}} \Rightarrow \underline{\underline{\sigma}} = -p\underline{\underline{1}}$	Thermo-Mechanical	6
	Constitutive Equations	6 eqns.

Introducing the constitutive equation in the Cauchy equation, we get the fundamental equation of hydrostatics:

$$\underline{\underline{\boldsymbol{\sigma}}} = -p\underline{\underline{\boldsymbol{1}}} \Rightarrow \vec{\boldsymbol{\nabla}} \cdot (-p\underline{\underline{\boldsymbol{1}}}) = -\vec{\boldsymbol{\nabla}}p_0 \rightarrow \begin{cases} -\vec{\boldsymbol{\nabla}}p_0 + \rho_0 \vec{\boldsymbol{b}} = \vec{\boldsymbol{0}} \\ \frac{\partial p_0}{\partial x_i} + \rho_0 b_i = 0; \quad i. \in \{1, 2, 3\} \end{cases}$$

We know the  $\rho_0$  we find the pressure p from above equation, then we define the stress  $\underline{\underline{\sigma}}$ 

# **Barotropic Perfect Fluids**

### Barotropic Perfect Fluids

A perfect fluid is a Newtonian fluid without viscosity ( $\mu = \lambda = 0$ ):

$$\underline{\underline{\sigma}} = -p\underline{\underline{1}} + \lambda \operatorname{Tr}(\underline{\underline{d}})\underline{\underline{1}} + 2\mu \underline{\underline{d}} \Rightarrow \underline{\underline{\sigma}} = -p\underline{\underline{1}} \Rightarrow \underline{\underline{\sigma}} = -p\underline{\underline{1}}$$

at hydrostatic stress state. Therefore:

$$\begin{split} \vec{\nabla} \cdot \underline{\underline{\sigma}} &= -\vec{\nabla} p \\ \underline{\underline{\sigma}} &: \underline{\underline{d}} &= -p \underline{\underline{1}} : \underline{\underline{d}} = -p \operatorname{Tr}(\underline{\underline{d}}) \end{split}$$

In a barotropic fluid temperature does not affect the kinetic state equation:

$$F(\rho, p) = 0 \Rightarrow \rho = \rho(p)$$

Note that most liquids can be assumed as barotropic (but not perfect). Some gases under certain circumstances.

### Barotropic Perfect Fluids: Field Equations

Hydrostatic problem  $\vec{v}(\vec{x},t) = \text{constant}$  is defined as:

	Conservation of Mass	1 egn.
$\dot{\rho} + \rho \vec{\nabla} \cdot \vec{v} = 0$		r cqii.
	Continuity Equation	
$\overrightarrow{m{ abla}} \cdot \underline{m{\sigma}} +  ho \vec{m{b}} =  ho \dot{\vec{m{v}}} \Rightarrow - \vec{m{\nabla}} p +  ho \vec{m{b}} =  ho \dot{\vec{m{v}}}$	Linear Momentum Balance	3 eqns.
	<b>Euler's Equation</b>	
$F(\rho, p) = 0 \Rightarrow \rho = \rho(p)$	Kinetic State Equation	1 eqn.

There are 5 scalar quantities unknowns:  $\rho$ ,  $\vec{v}$ , p

### Bernoilli's Trinomial

Consider a barotropic fluid under potential body forces:

$$\phi(\vec{\boldsymbol{x}},t) = gz \Rightarrow -\vec{\boldsymbol{\nabla}}\phi(\vec{\boldsymbol{x}},t) = -\begin{bmatrix} \frac{\partial\phi}{\partial x} & \frac{\partial\phi}{\partial y} & \frac{\partial\phi}{\partial z} \end{bmatrix}^T = \begin{bmatrix} 0 \\ 0 \\ -g \end{bmatrix}$$

The following lemmas hold:

▶ Lemma 1. For barotropic fluids there exist a function  $P(\vec{x},t) = \hat{P}(p(\vec{x},t))$  such that.

$$\vec{\boldsymbol{\nabla}}\boldsymbol{p} = \rho\vec{\boldsymbol{\nabla}}\boldsymbol{P}$$

▶ Lemma 2. The convective term of the acceleration can be expressed as:

$$ec{m{v}}\cdotec{m{
abla}}ec{m{v}}=2ec{m{\omega}} imesec{m{v}}+ec{m{
abla}}\left(rac{1}{2}v^2
ight)$$

where  $2\vec{\boldsymbol{\omega}} = \vec{\boldsymbol{\nabla}} \times \vec{\boldsymbol{v}}$  is the vorticity vector.



### Bernoilli's Trinomial

Using the Euler's equation:

$$-\vec{\nabla}p + \rho\vec{b} = \rho\vec{v} \rightarrow -\frac{1}{\rho}\vec{\nabla}p + \vec{b} = \frac{d\vec{v}}{dt}$$
$$\frac{1}{\rho}\vec{\nabla}p = \vec{\nabla}P; \vec{b} = \vec{\nabla}\phi$$
$$\vec{v}\vec{\nabla}\cdot\vec{v}$$

Combining the equations:

$$-\vec{m \nabla}P - \vec{m \nabla}\phi = rac{\partial ec{m v}}{\partial t} + 2ec{m \omega} imes ec{m v} + ec{m \nabla}\left(rac{1}{2}v^2
ight)$$

Finally we can derive the equation of motion for a barotropic perfect fluid:

$$-\vec{\nabla} \underbrace{\left[P + \phi + \frac{1}{2}v^2\right]}_{\text{Bernoulli's Trinomial}} = \frac{\partial \vec{v}}{\partial t} + 2\vec{\omega} \times \vec{v}$$

# **Newtonian Viscous Fluids**

# **Governing Equations**

The general fluid mechanics problem:

6			
$\dot{\rho} + \rho \vec{\nabla} \cdot \vec{v} = 0$	Conservation of Mass	1 eqn.	
$\rho + \rho \mathbf{v} \cdot \mathbf{v} = 0$	Continuity Equation		
$ec{m{ abla}}\cdot oldsymbol{ec{\sigma}} +  ho oldsymbol{ec{b}} =  ho \dot{ar{v}}$	Linear Momentum Balance	3 eqns.	
	Cauchy's Equation of motion		
$\rho \dot{u} = \underline{\underline{\boldsymbol{\sigma}}} : \underline{\underline{\boldsymbol{d}}} + \rho r - \vec{\boldsymbol{\nabla}} \cdot \vec{\boldsymbol{q}}$	Energy Balance	1 eqns.	
	First law of Thermodynamics		
$\underline{\underline{\boldsymbol{\sigma}}} = -p\underline{\underline{1}} + \lambda \operatorname{Tr}(\underline{\underline{\boldsymbol{d}}})\underline{\underline{1}} + 2\mu\underline{\underline{\boldsymbol{d}}}$	Mechanical	6 eqns.	
	Constitutive Equations		
$s = s(\underline{\boldsymbol{d}}, \theta, \rho)$	Entropy	1 eqn.	
$s=s(\underline{\underline{u}},  heta,  ho)$	Constitutive equation		
→ 12 <del>-7</del> 0	Thermal constitutive equation. Fourier	3 eqns.	
$ec{m{q}} = -K \vec{m{ abla}}  heta$	Law of conduction		
$u = u(\rho, \theta)$ $F(\rho, \theta, p) = 0$	Caloric and Kinetic State	2 eqns.	
17 scalar unknowns: $\rho(1), \vec{v}(3), \underline{\underline{\sigma}}(6), u(1), \vec{q}(3), \theta(1), s(1), p(1)$			

# Navier-Stokes equations

Consider the lemmas:

Lemma 1:

$$\vec{\nabla} \cdot \underline{\underline{d}} = \frac{1}{2} \Delta \vec{v} + \frac{1}{2} \vec{\nabla} (\vec{\nabla} \cdot \vec{v})$$

where  $\underline{d}$  is the deformation rate tensor.

Lemma 2:

$$\vec{\nabla} \cdot (a\underline{\mathbf{1}}) = \vec{\nabla} a$$

where  $a(\vec{x}, t)$  is a scalar fuction

Introducing the constitutive equation of stress in the term of Cauchy's equation (  $Tr(\underline{\underline{d}}) = \vec{\nabla} \cdot \vec{v}$ ):

$$\vec{\boldsymbol{\nabla}} \cdot \underline{\underline{\boldsymbol{\sigma}}} = \vec{\boldsymbol{\nabla}} \cdot (-p\underline{\underline{\mathbf{1}}} + \lambda \operatorname{Tr}(\underline{\underline{\boldsymbol{d}}})\underline{\underline{\mathbf{1}}} + 2\mu\underline{\underline{\boldsymbol{d}}}) = -\vec{\boldsymbol{\nabla}}p + \lambda \underbrace{\vec{\boldsymbol{\nabla}}(\operatorname{Tr}(\underline{\underline{\boldsymbol{d}}}))}_{\vec{\boldsymbol{\nabla}}(\vec{\boldsymbol{\nabla}} \cdot \vec{\boldsymbol{v}})} + \mu \Delta \vec{\boldsymbol{v}} + \mu \vec{\boldsymbol{\nabla}}(\vec{\boldsymbol{\nabla}} \cdot \vec{\boldsymbol{v}})$$

### Navier-Stokes equations

The linear momentum balance equation can be arranged as follows:

$$\vec{\nabla} \cdot \underline{\underline{\sigma}} + \rho \vec{b} = \rho \dot{\vec{v}} \Rightarrow -\vec{\nabla} p + (\lambda + \mu) \vec{\nabla} (\vec{\nabla} \cdot \vec{v}) + \mu \Delta \vec{v} + \rho \vec{b} = \rho \frac{d\vec{v}}{dt}$$

Finally the NAVIER-STOKES equations can be written as:

$$-\vec{\nabla}p + (\lambda + \mu)\vec{\nabla}(\vec{\nabla} \cdot \vec{v}) + \mu\Delta\vec{v} + \rho\vec{b} = \rho\frac{d\vec{v}}{dt}$$
$$-\frac{\partial p}{\partial x_i} + (\lambda + \mu)\frac{\partial^2 v_j}{\partial x_i \partial x_j} + \mu\frac{\partial^2 v_i}{\partial x_i \partial x_j} + \rho b_i = \rho\frac{dv_i}{dt}, \quad 1, j \in \{1, 2, 3\}$$

The Navier-Stokes equations are the equation of motion (balance of linear momentum) written in terms of velocities.

There are 4 unknowns:  $\vec{v}(3)$ , p(1) and 3 equations.

For incompressible fluids the  $(\vec{\nabla}\cdot\vec{\pmb{v}})=0$  and  $\underline{\underline{\pmb{\sigma}}}=-p\underline{\underline{\pmb{1}}}+2\mu\underline{\underline{\pmb{d}}}$ 



### **Energy Equations**

The energy balance equation can be given as, [1]:

$$\begin{split} \rho\frac{du}{dt} &= -p\vec{\nabla}\cdot\vec{v} + \rho\ r + \vec{\nabla}\cdot(K\vec{\nabla}\theta) + \underbrace{\kappa\,Tr^2(\underline{\underline{d}}) + 2\mu\underline{\underline{d}}_{dev}:\underline{\underline{d}}_{dev}:\underline{\underline{d}}_{dev}}_{\text{Dissipative Power }2W_D} \\ \rho\frac{du}{dt} &= -p\frac{\partial v_i}{\partial x_i} + \rho\ r + \frac{\partial}{\partial x_i}(K\frac{\partial\theta}{\partial x_i}) + \kappa\left(\frac{\partial v_i}{\partial x_i}\right)^2 + 2\mu\,d_{ij}^{dev}d_{ij}^{dev}; \quad i,j\in\{1,2,3\} \end{split}$$

The energy equation is just the energy balance in terms of velocity and pressure.

### References I



X. Oliver and C. Agelet de Saracibar.

Continuum Mechanics for Engineers. Theory and Problems.
2017.