

Nguyen Xuan Binh 887799 Assignment Week 5

Exercise 1: For an isotropic linear elastic material with modulus of elasticity equal to $E = 71 \text{ GPa}$, shear modulus $G = 26.6 \text{ GPa}$. Determine the strain tensor components and the internal energy density at the point with stress: $\begin{bmatrix} 20 & -4 & 5 \\ -4 & 0 & 10 \\ 5 & 10 & 15 \end{bmatrix} \text{ MPa}$

a) Strain tensor components:

$$\text{We have: } \mu = G = \frac{E}{2(1+\nu)} \Rightarrow 2(1+\nu) = \frac{E}{G} = \frac{71}{26.6} = \frac{355}{133} \Rightarrow \nu = \frac{89}{266}$$

\Rightarrow Strain tensor components:

$$\epsilon_x = \frac{1}{E} (\sigma_x - \nu(\sigma_y + \sigma_z)) = \frac{1}{71 \cdot 10^3} (20 - \frac{89}{266}(0 + 15)) \approx 211 \cdot 10^{-6}$$

$$\epsilon_y = \frac{1}{E} (\sigma_y - \nu(\sigma_x + \sigma_z)) = \frac{1}{71 \cdot 10^3} (0 - \frac{89}{266}(20 + 15)) \approx -165 \cdot 10^{-6}$$

$$\epsilon_z = \frac{1}{E} (\sigma_z - \nu(\sigma_x + \sigma_y)) = \frac{1}{71 \cdot 10^3} (15 - \frac{89}{266}(20 + 0)) \approx 117 \cdot 10^{-6}$$

$$\gamma_{yz} = \frac{1}{G} (T_{yz}) = \frac{1}{26.6 \cdot 10^3} \cdot 10 = 376 \cdot 10^{-6}$$

$$\gamma_{xz} = \frac{1}{G} (T_{xz}) = \frac{1}{26.6 \cdot 10^3} \cdot 5 = 188 \cdot 10^{-6}$$

$$\gamma_{xy} = \frac{1}{G} (T_{xy}) = \frac{1}{26.6 \cdot 10^3} \cdot (-4) = -150.4 \cdot 10^{-6}$$

$$\Rightarrow \text{Strain tensor: } 10^{-6} \begin{bmatrix} 211 & -75.2 & 94 \\ -75.2 & -165 & 188 \\ 94 & 188 & 117 \end{bmatrix} = \underline{\underline{\epsilon}}$$

b) Internal energy density:

$$\hat{u}(\underline{\underline{\epsilon}}) = \frac{1}{2} \underline{\underline{\sigma}} : \underline{\underline{\epsilon}} = \frac{1}{2} \left(20 \times 211 \cdot 10^{-6} + 0 \times (-165 \cdot 10^{-6}) + 15 \times (117 \cdot 10^{-6}) \right. \\ \left. + 10 \times 376 \cdot 10^{-6} + 5 \times 188 \cdot 10^{-6} + (-4) \times (-150.4 \cdot 10^{-6}) \right)$$

$$\Rightarrow \hat{u}(\underline{\underline{\epsilon}}) = 5638.3 \times 10^{-6}$$

Exercise 2:

a) We have $V_{\text{mercury}} = 0.004 \text{ m}^3 \Rightarrow h_{\text{mercury}} = \frac{0.004}{0.1 \times 0.1} = 0.4 \text{ m}$

\Rightarrow Height that reach the mercury: $H = h_{\text{mercury}} + [h_{\text{rubber0}} + h_{\text{rubber0}} \varepsilon_z]$ (Solve in b)

b) Obtain the stress state at any point of the rubber block

Weight of the mercury: $\rho V = 13580 \times 0.004 = 54.32 \text{ (kg)}$

\Rightarrow Weight of the mercury: $mg = 54.32 \times 10 = 543.2 \text{ (N)}$

Cross sectional area: $0.1 \times 0.1 = 0.01 \text{ m}^2$

\Rightarrow The pressure of the mercury on the rubber is $\frac{-543.2}{0.01} = -54320 \text{ Pa}$

Since the only force that acts on the rubber is the mercury's weight along the z direction and the rubber does not deform in x, y direction

$$\Rightarrow \sigma_z = -54320 \text{ Pa}, \varepsilon_x = \varepsilon_y = 0$$

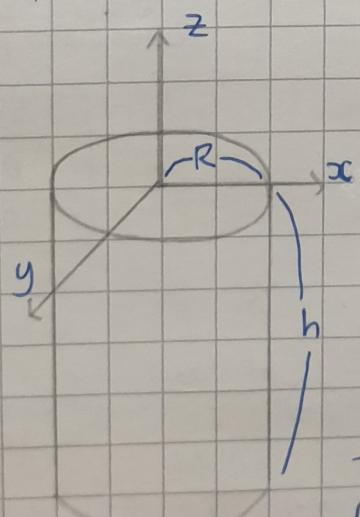
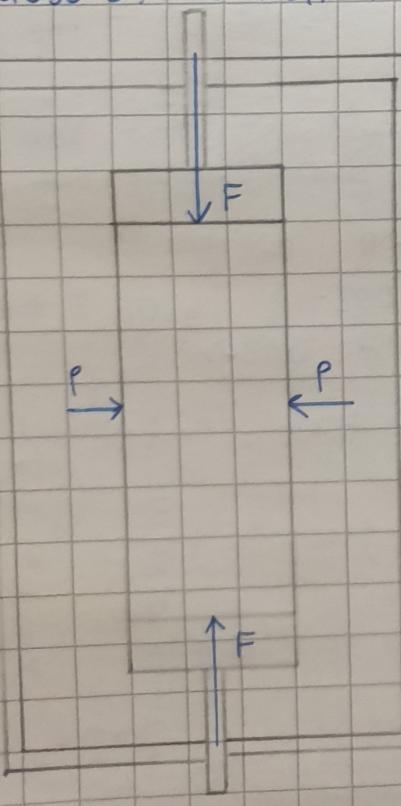
$$\Rightarrow \varepsilon_{xz} = \frac{1}{E} (\sigma_x - v(\sigma_y + \sigma_z)) \Rightarrow 0 = \frac{1}{2.94 \times 10^6} (\sigma_x - v(\sigma_y - 54320)) \quad \begin{matrix} 0.4 \\ m \end{matrix}$$

$$\Rightarrow \varepsilon_y = \frac{1}{E} (\sigma_y - v(\sigma_x + \sigma_z)) \Rightarrow 0 = \frac{1}{2.94 \times 10^6} (\sigma_y - v(\sigma_x - 54320)) \quad \begin{matrix} -L \\ 0.1m \end{matrix}$$

$$\Rightarrow \varepsilon_z = \frac{1}{E} (\sigma_z - v(\sigma_x + \sigma_y)) \Rightarrow \sigma_x = \sigma_y = -6035.5 \text{ Pa}, \varepsilon_z = -18 \times 10^{-3} \quad \begin{matrix} 0.1m \\ \downarrow \end{matrix}$$

$$\Rightarrow \text{Stress state at any point: } \underline{\underline{\sigma}} = \begin{bmatrix} -6035.5 & 0 & 0 \\ 0 & -6035.5 & 0 \\ 0 & 0 & -54320 \end{bmatrix} \quad \begin{matrix} \text{and the height that reach the mercury} \\ H = 0.4 + (0.5 - 0.5 \times 18 \times 10^{-3}) \\ = 0.891 \text{ m} \end{matrix}$$

Exercise 3: Visualization



$$R = 0.05 \text{ m} \quad h = 0.25 \text{ m} \quad E = 3 \times 10^4 \quad v = 0.2$$

$$p = 15 \text{ MPa} \quad F = 2.35619 \times 10^5 \text{ N}$$

a) Obtain the stress components
Cross sectional area

$$A = \pi R^2 = \pi (0.05)^2 = \frac{1}{400} \pi (\text{m}^2)$$

Stress along the z axis

$$\sigma_z = -\frac{F}{A} = -\frac{2.35619 \times 10^5}{1/400 \pi}$$

$$\Rightarrow \sigma_z = -29.999942 \times 10^6 \text{ Pa} \\ = -29.999942 \text{ MPa}$$

The cylinder is in equilibrium $\Rightarrow x$ axis and y -axis has stress of p

$$\Rightarrow \underline{\underline{\sigma}} = \begin{bmatrix} -15 & 0 & 0 \\ 0 & -15 & 0 \\ 0 & 0 & -29.99 \end{bmatrix} \text{ MPa.}$$

b) The strain tensor components

$$\varepsilon_x = \frac{1}{E} (\sigma_x - \nu(\sigma_y + \sigma_z)) = \frac{1}{3 \times 10^4} (15 - 0.2(15 + 29.99)) = 200 \times 10^{-6}$$

$$\varepsilon_y = \frac{1}{E} (\sigma_y - \nu(\sigma_x + \sigma_z)) = \frac{1}{3 \times 10^4} (15 - 0.2(15 + 29.99)) = 200 \times 10^{-6}$$

$$\varepsilon_z = \frac{1}{E} (\sigma_z - \nu(\sigma_x + \sigma_y)) = \frac{1}{3 \times 10^4} (29.99 - 0.2(15 + 15)) = 799.6 \times 10^{-6}$$

$$\gamma_{xy} = \gamma_{yz} = \gamma_{xz} = 0$$

$$\Rightarrow \underline{\underline{\varepsilon}} = 10^{-6} \begin{bmatrix} 200 & 0 & 0 \\ 0 & 200 & 0 \\ 0 & 0 & 799.6 \end{bmatrix}$$

c) Obtain the displacement field components

$$\frac{\partial u_x}{\partial x} = \varepsilon_x \Rightarrow u_x = 200 \times 10^{-6} x$$

$$\frac{\partial u_y}{\partial y} = \varepsilon_y \Rightarrow u_y = 200 \times 10^{-6} y$$

$$\frac{\partial u_z}{\partial z} = \varepsilon_z \Rightarrow u_z = 799.6 \times 10^{-6} z$$

$$\Rightarrow \text{Displacement field} : \begin{bmatrix} u_x \\ u_y \\ u_z \end{bmatrix} = 10^{-6} \begin{bmatrix} 200x \\ 200y \\ 799.6z \end{bmatrix}$$

Exercise 4: The stress state at a point of a body made out of isotropic linear elastic material is given by $\begin{bmatrix} 6 & 2 & 0 \\ 2 & -3 & 0 \\ 0 & 0 & 0 \end{bmatrix} \text{ MPa}$ with $E = 207 \text{ GPa}$, $G = 80 \text{ GPa}$

$$\Rightarrow \nu = \frac{E}{2G} - 1 = \frac{57}{160}$$

a) Determine the strain tensor components

$$\varepsilon_x = \frac{1}{E} (\sigma_x - \nu(\sigma_y + \sigma_z)) = \frac{1}{207 \times 10^3} (6 - \frac{57}{160}(-3 + 0)) = 33.24 \times 10^{-6}$$

$$\varepsilon_y = \frac{1}{E} (\sigma_y - \nu(\sigma_x + \sigma_z)) = \frac{1}{207 \times 10^3} (-3 - \frac{57}{160}(6 + 0)) = -23 \times 10^{-6}$$

$$\varepsilon_z = \frac{1}{E} (\sigma_z - \nu(\sigma_x + \sigma_y)) = \frac{1}{207 \times 10^3} (0 - \frac{57}{160}(6 - 3)) = -4.26 \times 10^{-6}$$

$$\gamma_{yz} = \frac{1}{G} T_{yz} = \frac{1}{80 \times 10^3} \cdot 0 = 0 \quad \gamma_{xz} = \frac{1}{G} T_{xz} = \frac{1}{80 \times 10^3} \cdot 0 = 0$$

$$\gamma_{xy} = \frac{1}{G} T_{xy} = \frac{1}{80 \times 10^3} \cdot 2 = 25 \times 10^{-6}$$

$$\Rightarrow \underline{\underline{\epsilon}} = 10^{-6} \begin{bmatrix} 33.24 & 12.5 & 0 \\ 12.5 & -23 & 0 \\ 0 & 0 & -4.26 \end{bmatrix}$$

b) Consider a cube of side 5 cm is subjected to this stress state. Define the volume variation

$$\text{Volume of the cube: } V_0 = 5^3 = 125 \text{ cm}^3$$

$$\text{Volume variation} = \text{volumetric strain} \times V_0 \Rightarrow \Delta V = \epsilon_V \times V_0$$

$$\text{for small deformation: } \epsilon_V = \epsilon_x + \epsilon_y + \epsilon_z = 10^{-6} (33.24 - 23 - 4.26) \\ = 5.98 \times 10^{-6}$$

$$\Rightarrow \text{Volume variation: } \Delta V = \epsilon_V \times V_0 = 5.98 \times 10^{-6} \times 125 = 747.5 \times 10^{-6} \text{ cm}^3$$