

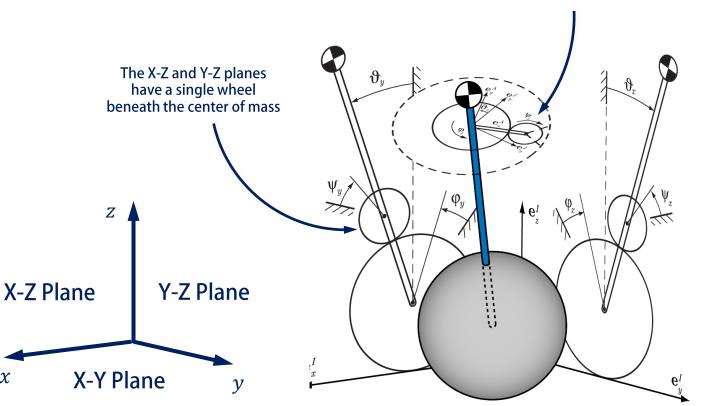
#### **ROB 311 – Lecture 14**

- Today:
  - Review torque conversion
  - Discuss kinematic conversions
  - Finish lab exercises (second half+)

- Announcements
  - HW 3 posted, due 10/20 at class start
  - Stay tuned for HW / quiz solutions
  - No class Tuesday (10/18)! Fall study break
  - Midterm exam 11/8

### **Full Planar Model**

The X-Y plane contains a single wheel that rotates the ball-bot



 $\vartheta_{axis}$  Body rotation

 $\Psi_{axis}$  Wheel rotation

 $\varphi_{axis}$  Ball rotation

Since we want to balance and drive, the rotation of the ball (i.e. X-Y plane) is mostly irrelevant

- Each plane has two DOFs: Rotations of the chassis and ball
- X-Z and Y-Z planes are required for balance and motion—we will focus on these
- The torques and motion of the virtual wheels are different from the real wheels

### Virtual and Real Wheel Contact

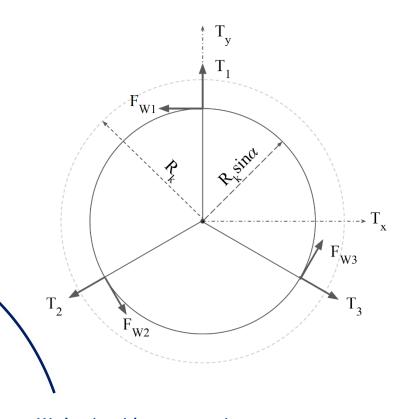
Torque on the wheel - cross product

$$T_{Wi} = r_{Wi} \times F_{Wi}$$
 for  $i = 1,2,3$   
 $T_{Wj} = r_{Wj} \times F_{Wj}$  for  $j = x, y, z$ 

Virtual wheel contact points in (X-Z and Y-Z planes)

$$r_{Wx} = R_k \begin{bmatrix} 0 \\ 0 \\ 1 \end{bmatrix}$$
  $r_{Wy} = R_k \begin{bmatrix} 0 \\ 0 \\ 1 \end{bmatrix}$   $r_{kWz} = R_k \begin{bmatrix} 0 \\ \sin(\alpha) \\ 0 \end{bmatrix}$ 

$$r_{kWz} = R_k \begin{bmatrix} 0\\ \sin(\alpha)\\ 0 \end{bmatrix}$$



We begin with contact points then discuss forces

Real wheel contact points

$$r_{W1} = R_k \begin{bmatrix} 0 \\ \sin(\alpha) \\ \cos(\alpha) \end{bmatrix} \qquad r_{W2} = R_k \begin{bmatrix} \frac{-\sqrt{3}}{2} \sin(\alpha) \\ \frac{1}{2} \sin(\alpha) \\ \cos(\alpha) \end{bmatrix} \qquad r_{W3} = R_k \begin{bmatrix} \frac{\sqrt{3}}{2} \sin(\alpha) \\ \frac{1}{2} \sin(\alpha) \\ \cos(\alpha) \end{bmatrix}$$

$$r_{W3} = R_k \begin{bmatrix} \frac{\sqrt{3}}{2} \sin(\alpha) \\ -\frac{1}{2} \sin(\alpha) \\ \cos(\alpha) \end{bmatrix}$$

### **Virtual and Real Forces**

Torque on the wheel - cross product

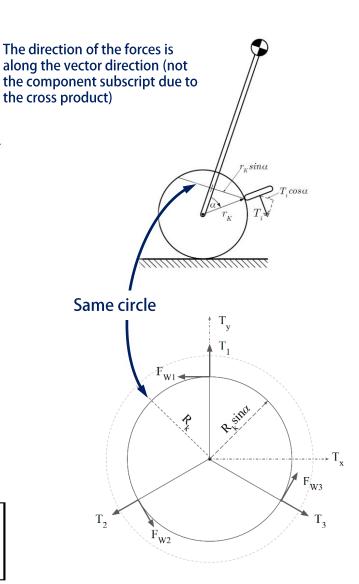
$$T_{Wi} = r_{Wi} \times F_{Wi}$$
 for  $i = 1,2,3$   
 $T_{Wj} = r_{Wj} \times F_{Wj}$  for  $j = x, y, z$ 

Virtual wheel forces

$$F_{Wx} = \frac{T_x}{r_w} \begin{bmatrix} 0 \\ 1 \\ 0 \end{bmatrix} \qquad F_{Wy} = \frac{T_y}{r_w} \begin{bmatrix} 1 \\ 0 \\ 0 \end{bmatrix} \qquad F_{Wz} = \frac{T_z}{r_w} \begin{bmatrix} -1 \\ 0 \\ 0 \end{bmatrix}$$

Real wheel forces

$$F_{W1} = \frac{T_1}{r_w} \begin{bmatrix} -1\\0\\0 \end{bmatrix} \qquad F_{W2} = \frac{T_2}{r_w} \begin{bmatrix} 1/2\\-\sqrt{3}/2\\0 \end{bmatrix} \qquad F_{W3} = \frac{T_3}{r_w} \begin{bmatrix} 1/2\\\sqrt{3}/2\\0 \end{bmatrix}$$



## **Relating Virtual and Real Torques**

- Torque in both coordinate systems is conserved
  - Not forces conserved—this says the torques around the ball will be equivalent

$$T_1 + T_2 + T_3 = T_x + T_y + T_z$$

Written in terms of force and perpendicular distance

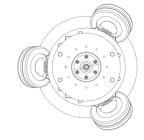
$$r_{W1} \times F_{W1} + r_{W2} \times F_{W2} + r_{W3} \times F_{W3} \dots$$
  
=  $r_{Wx} \times F_{Wx} + r_{Wy} \times F_{Wy} + r_{Wz} \times F_{Wz}$ 

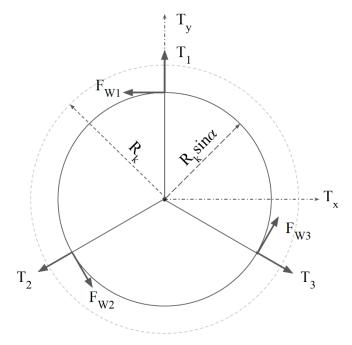
Solving for torques

$$T_1 = \frac{1}{3} \left( T_z - \frac{2T_y}{\cos(\alpha)} \right)$$

$$T_2 = \frac{1}{3} \left( T_z + \frac{1}{\cos(\alpha)} \left( -\sqrt{3}T_x + T_y \right) \right)$$

$$T_3 = \frac{1}{3} \left( T_z + \frac{1}{\cos(\alpha)} \left( \sqrt{3}T_x + T_y \right) \right)$$









# **Relating Virtual and Real Torques**

- $\alpha$   $\pi/_4$  or 45°
- $\beta$   $\pi/_2$  or 90°

- Conversion equations provided in ETH Zurich dissertation
- Solved for real motor torques

$$T_{1} = \frac{1}{3} \cdot \left( T_{z} + \frac{2}{\cos \alpha} \cdot (T_{x} \cdot \cos \beta - T_{y} \cdot \sin \beta) \right)$$

$$T_{2} = \frac{1}{3} \cdot \left( T_{z} + \frac{1}{\cos \alpha} \cdot \left( \sin \beta \cdot (-\sqrt{3}T_{x} + T_{y}) - \cos \beta \cdot (T_{x} + \sqrt{3}T_{y}) \right) \right)$$

$$T_{3} = \frac{1}{3} \cdot \left( T_{z} + \frac{1}{\cos \alpha} \cdot \left( \sin \beta \cdot (\sqrt{3}T_{x} + T_{y}) + \cos \beta \cdot (-T_{x} + \sqrt{3}T_{y}) \right) \right)$$

These are the same equations as the previous slide, but written generally for any  $\alpha$  and  $\beta$ 

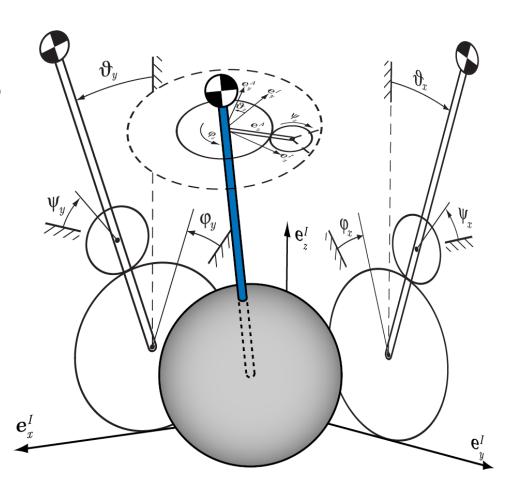
Solved for virtual motor torques

$$T_x = \cos \alpha \cdot \left( T_1 \cdot \cos \beta - T_2 \cdot \sin(\beta + \frac{\pi}{6}) + T_3 \cdot \sin(\beta - \frac{\pi}{6}) \right)$$

$$T_y = \cos \alpha \cdot \left( -T_1 \cdot \sin \beta - T_2 \cdot \cos(\beta + \frac{\pi}{6}) + T_3 \cdot \cos(\beta - \frac{\pi}{6}) \right)$$

$$T_z = \underline{T_1 + T_2 + T_3}$$

- We now can describe torque from the wheels to full planar model
  - Go from x and y components to the 3 motor torques
- Can we develop a similar analysis for velocity transformation?
- The real and virtual wheels are the same size
- Do the real wheels and virtual wheels have the same velocity?
  - No, wheel orientation affects velocity
  - Wheels only spin with the component of linear velocity perpendicular to wheel axis



- Planar model has two DOFs  $\varphi_{axis}$  and  $\vartheta_{axis}$
- Lets look at the velocities of points A, B, and C

$$v_{Ay} = 0$$

$$v_{Az} = 0$$

$$v_{By} = \dot{y}_k + \dot{\varphi}_x R_k \cos(\theta_x) \qquad \text{Derive but a}$$

$$v_{Bz} = -\dot{\varphi}_x R_k \sin(\theta_x)$$

$$v_{Cy} = \dot{y}_k + \left( (R_k + R_w)\dot{\theta}_x + \dot{\psi}_x R_w \right) \cos(\theta_x)$$

$$v_{Cz} = -\left( (R_k + R_w)\dot{\theta}_x + \dot{\psi}_x R_w \right) \sin(\theta_x)$$

$$v_{By} = v_{Cy}$$

$$v_{Bz} = v_{Cz}$$

 $v_{A-Ccomponent}$ 

Velocity component of point A, B, or C

 $\varphi_{axis}$ 

Ball angle

 $\vartheta_{axis}$ 

**Body** angle

 $\Psi_{axis}$ 

Wheel angle

 $y_k$ 

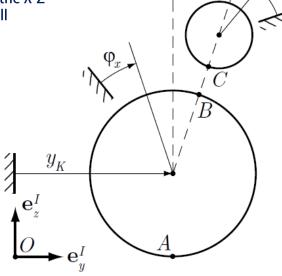
Linear position of ball center

Time derivative denoted by dot

operator

Y-Z Plane

Derived for the Y-Z plane but applicable to the X-Z plane as well



We can set the velocities equal at points A and B

$$v_{A-Ccomponent}$$

Velocity component of point A, B, or C

 $\varphi_{axis}$ 

**Ball angle** 

 $\vartheta_{axis}$ 

**Body angle** 

 $\Psi_{axis}$ 

Wheel angle

 $y_k$ 

Linear position of ball center

Time derivative denoted by dot operator

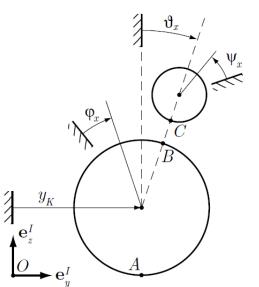
$$v_{By} = v_{Cy}$$

$$v_{Bz} = v_{Cz}$$

$$v_y = \dot{y}_k + \dot{\varphi}_x R_k \cos(\theta_x) = \dot{y}_k + ((R_k + R_w)\dot{\theta}_x + \dot{\psi}_x R_w)\cos(\theta_x)$$

$$v_z = -\dot{\varphi}_x R_k \sin(\theta_x) = -((R_k + R_w)\dot{\theta}_x + \dot{\psi}_x R_w)\sin(\theta_x)$$

$$\dot{\varphi}_{x}R_{k} = (R_{k} + R_{w})\dot{\vartheta}_{x} + \dot{\psi}_{x}R_{w}$$



 We want to know how fast the wheels need to spin for known / induced ball and body rotations

- Planar model has two DOFs  $\varphi_{axis}$  and  $\vartheta_{axis}$
- Lets look at the velocities of points A, B, and C

$$\dot{\varphi}_{x}R_{k} = (R_{k} + R_{w})\dot{\vartheta}_{x} + \dot{\psi}_{x}R_{w}$$
 
$$\dot{\psi}_{x} = \frac{R_{k}}{R_{w}}\dot{\varphi}_{x} - \frac{R_{k} + R_{w}}{R_{w}}\dot{\vartheta}_{x}$$
 
$$\dot{\psi}_{x} = \frac{R_{k}}{R_{w}}\left(\dot{\varphi}_{x} - \dot{\vartheta}_{x}\right) - \dot{\vartheta}_{x}$$
 
$$\dot{\psi}_{x} = \frac{R_{k}}{R_{w}}\left(\dot{\varphi}_{x} - \dot{\vartheta}_{x}\right) - \dot{\vartheta}_{x}$$
 Virtual wheel angular velocity 
$$\dot{\varphi}_{x} = \frac{R_{k}}{R_{w}}\left(\dot{\varphi}_{x} - \dot{\vartheta}_{x}\right) - \dot{\vartheta}_{x}$$

 $v_{A-Ccomponent}$  Velocity C

Velocity component of point A, B, or C

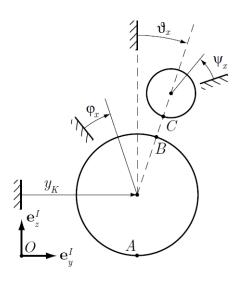
 $\phi_{axis}$  Ball angle

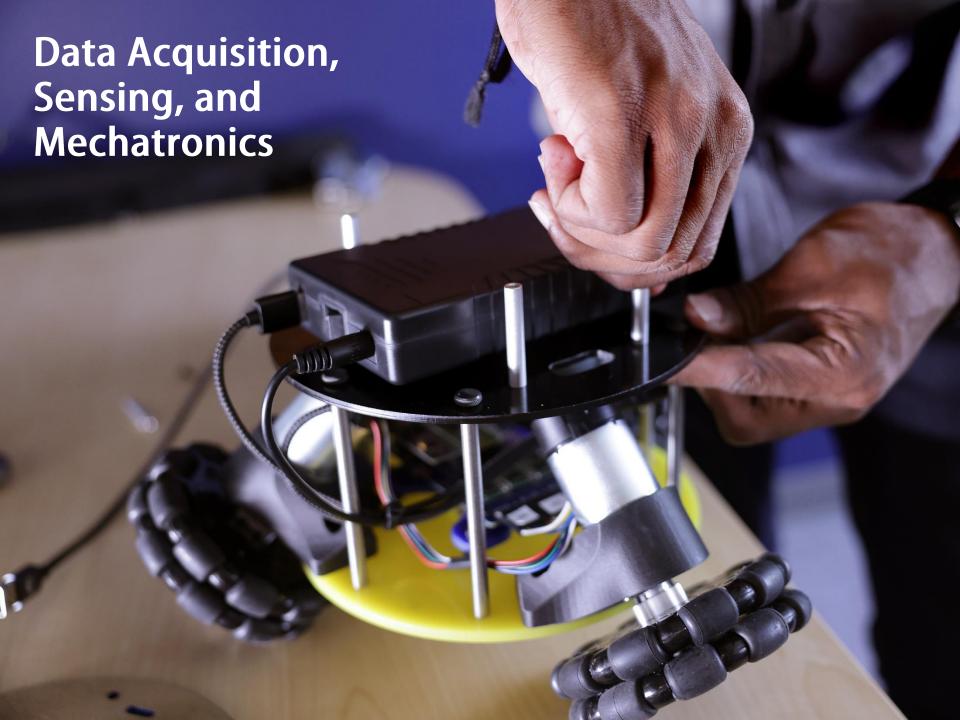
 $\vartheta_{axis}$  Body angle

 $\psi_{axis}$  Wheel angle

 $y_k$  Linear position of ball center

Time derivative denoted by dot operator





## **Data Acquisition**

- A key aspect of developing a robotic system is sensing the robot state / environment
- For the ball bot, we will want to know
  - Lean angle
  - Ball rotation (obtained from motor rotation)
- In the coming slides, we will learn how use sensors for data acquisition
- Two types of data acquisition
  - Analog: continuous voltages read by a computer and interpreted from the voltage value
  - Digital: information is sent over a communication bus in the form of 1s and 0s that contain the sensor data that has been digitized
- Our ball-bot design only uses digital sensors but we will quickly review analog sensing
  - Previously more common but becoming less common

### **Data Acquisition**

- Typically, sensors output voltage, which needs to be acquired by a digital computer for analysis and control
- Accomplished using a data acquisition system



- Analog to digital (A2D) converter: digitizes voltage to be read by a computer
- Three key attributes
  - Bits describes how many 'bins' can the voltage be separated into
  - Sample rate (Hz) how fast the loop runs / A2D converter is sampled
  - Input range (V) the total range of voltages able to be sampled

Resolution 
$$\longrightarrow \frac{\Delta V}{bit} = \frac{V_{range}}{2^{bits} - 1}$$

## **Analog Data Acquisition**

- To calculate the voltage, we sample the A2D system, which will return an integer number of bits
- These bits need to be converted back to voltage
- After converting to volts, it needs to be converted to the original units
  - This is accomplished using the sensors calibration curve or 'sensitivity'

$$V_{sensor} = V_{min} + \left(b \cdot \frac{V_{range}}{2^{bits} - 1}\right)$$

Number of bits returned from your A2D card

- Example your 16 bit DAQ system records a value of 11564, with an input range of -10 to 10 V.
- What is the voltage?

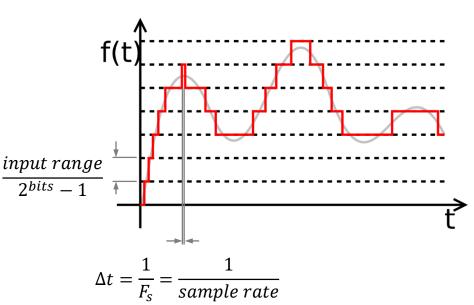
$$V = -10 + \left(11564 \cdot \frac{20}{65535}\right) = -6.47 V$$



# **Analog Data Acquisition**

- Signals are sampled periodically, at a frequency governed by the sample rate
- This causes some 'quantization' of the analog signal
- The sample rate is a key factor of a data acquisition system
- It governs the frequency content of what can be measured
- Higher sample rates can sense higher frequencies
- Nyquist criterion:

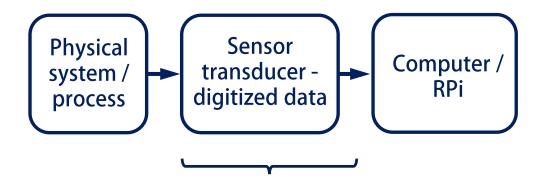
Maximum measurable 
$$f_{max} = \frac{1}{2}F_{s}$$



- The highest frequency that can be measured is half the sample rate
- Sample rate / loop rate must be set carefully to make sure the relevant frequencies can be recorded

## **Digital Data Acquisition**

- Digital sensors are becoming increasingly common
- Data are transmitted using digital communication (USB, I<sup>2</sup>C, etc.)
- No analog voltage concepts of resolution change for these sensors
  - Governed directly from the sensor, not the A2D card
- Nyquist frequency still applies
- Sampling still applies



Digital sensors communicate directly with acquisition computer

# **Digital Data Acquisition**

- Digital sensors are defined by their bit-level resolution
  - For example, 14-bit encoder (rotation sensor)

$$\frac{\circ}{bit} = \frac{360^{\circ}}{2^{14} - 1} = 0.0220$$





- Digital communication busses
  - USB
  - Inter-Integrated Circuit communication (I<sup>2</sup>C)

    Most common (for now)
  - Serial Peripheral Interface (SPI)
  - RS-232 (old school, serial port)
  - A program is used to request and interpret information from the sensor
- Information on communication and interpretation is provided in the sensor datasheet
- This program is called a 'driver' or 'API' and is usually written in C or Python

#### **ROB 311 – Lab 7**

- There is some spread in the ball-bot readiness between teams (this is okay)
- We will break here to allow some of the class to finish Lab 7
- Git
  - Revisit forking repositories
  - Authentication
  - Saving data



- Looping and indexing
- Importing packages
- Creating functions
- Saving data



