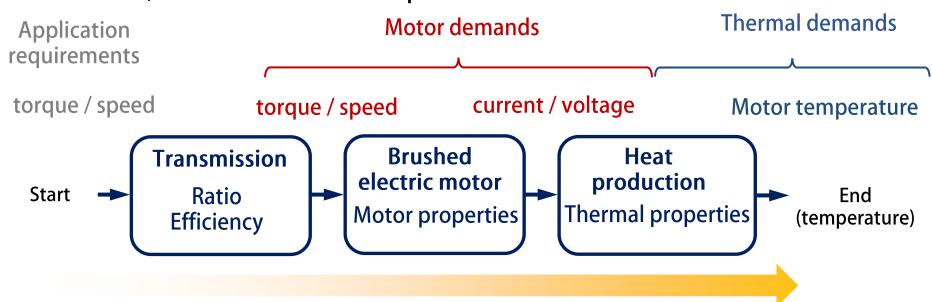


#### **ROB 311 – Lecture 2**

- Welcome to ROB 311!
- Today:
  - Review design framework
  - Discuss the initial blocks in the flow chart (transmissions and motors)
  - Discuss governing equations for brushed dc motors
  - Do a simple example and an example using the ball-bot

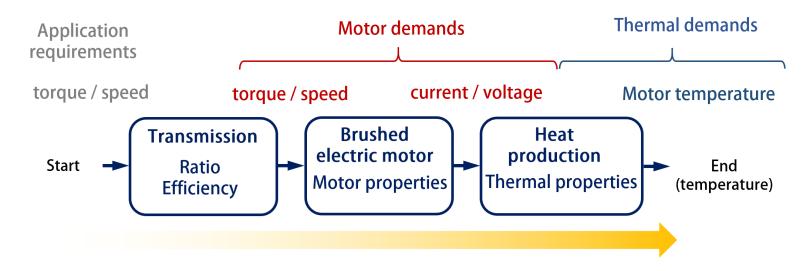
#### **Design Analysis Framework**

- As a reminder,
  - We have a description of the required torque to roll the ball / fight friction
  - We have a max velocity goal, say 1 m/s
- We use this torque and velocity to understand the demand on the motion, structural, and mechatronics components



Framework outcome: A motor that has the desired operating voltage, that is able to complete the task without overheating

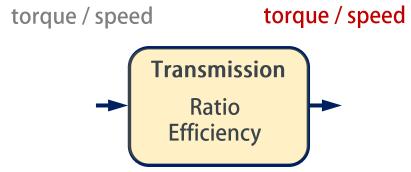
#### **Design Analysis Framework**



- What constraints might we have on the design, other than torque-velocity?
  - Battery voltage we will operate on 12 V
  - Max current the motor drivers can take 1-2 A continuous
- A motor-transmission combination is viable if
  - The required current and voltage demanded are within the constraints
  - The rise in motor temperature is acceptable

## **Modeling Transmissions**

- Lets begin with the first block transmissions
- Transmission exchange effort (torque or force) for flow (velocity)
- Transmissions convert one torque-speed regime to another torque-speed regime



- It is parameterized by two coefficients the ratio and efficiency
- In an ideal transmission, power is conserved, and efficiency is 100%



# **Modeling Transmissions**

- It is parameterized by two coefficients the ratio and efficiency
- In an ideal transmission, power is conserved, and efficiency is 100%

Velocity = 
$$\dot{\Psi}$$
  
Torque =  $T_w$ 

Ratio =  $N$ 

Velocity =  $\dot{\Psi} \cdot N$ 

Torque =  $\frac{T_w}{N}$ 

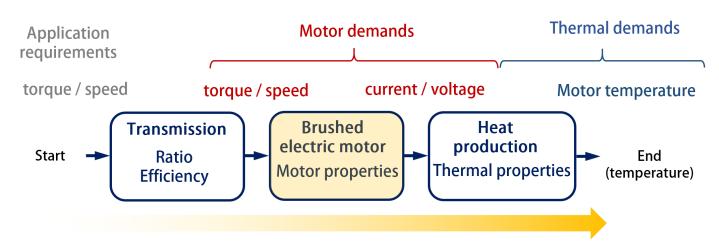
- Lets think about what transmissions do—why have them?
  - They trade torque for speed, which can be advantageous
- Why does a transmission on a bicycle matter so much?
  - The efficiency of the human body depends on:
    - Posture and speed
- We need our model of a transmission to account for inefficiency
  - Simplified models of inefficiency include it as a constant (0% 100%)



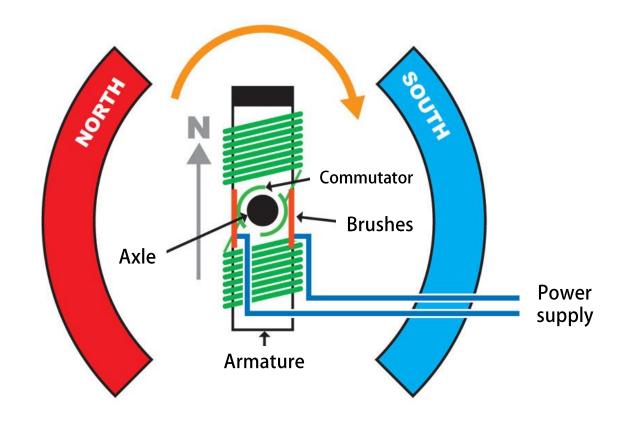
# **Modeling Transmissions**



- Inefficiency reduces the torque transferred through the transmission
- Common values of  $\eta$  are 0.8 0.95, depending on type / stages
- At this point, if we know the torque and velocity requirements, we can predict how these requirement are seen at the motor (after the transmission)



- This brings us to modeling brushed dc electric motors
- Our goal is to use the required torque-velocity at the motor input to determine the required current-voltage that must be provided by the power supply
- How do brushed motors work?



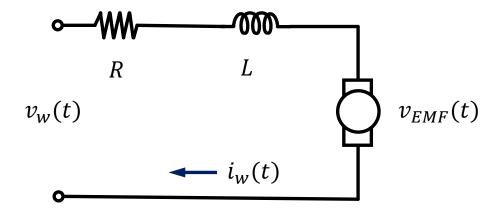
- Brushed motor considerations:
  - Brushes make mechanical contact to provide electrical power
  - Pros: Simple to implement
  - Cons:
    - Greater friction
    - Lower speeds
    - Result: Lower power / size → lower specific power (W/kg)
- Brushless motors exist that eliminate the need for mechanical brushes
  - Brushless motors have three windings (instead of one)
  - Brushless motors "spin" the current in the windings instead of the rotor
  - They require a computer to spin the currents in windings, known as a "brushless motor drive"
  - Higher speeds meaning higher specific power (W/kg)

- Which is right for your application?
- We quantify required torque / speed and current / voltage to make decision
- Typically assessed in tandem with the transmission ratio





- DC brushed motors
  - Rules of thumb:
    - Voltage is proportional to speed (effort)
    - Current is proportional to torque (flow)



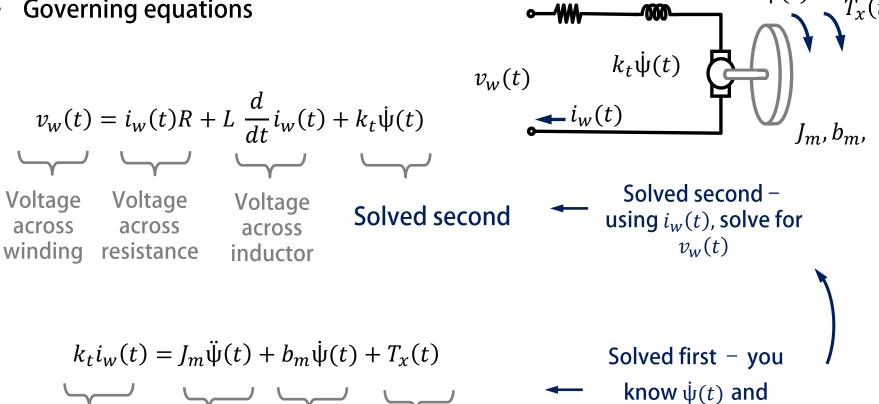
 Think of a motor as a transformer – it transforms power in the form of current and voltage to power in the form of current and torque (plus loss as heat)

derivatives → solve

for  $i_w(t)$ 

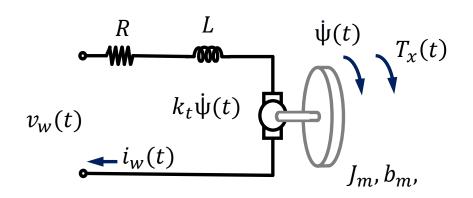
# **Modeling Brushed DC Motors**

**Governing equations** 



Exercise

$$v_w(t) = i_w(t)R + L \frac{d}{dt}i_w(t) + k_t\dot{\psi}(t)$$
$$k_ti_w(t) = J_m\ddot{\psi}(t) + b_m\dot{\psi}(t) + T_x(t)$$



Compute voltage and current at <u>steady state</u>:

$$i_W(t) = \frac{0.02 \cdot 20 + 2}{0.1} = 24 A$$

$$v_w(t) = 24 \cdot 0.2 + 0.1 \cdot 20 = 6.8 V$$

$$R = 0.2 \Omega$$

$$k_t = 0.1 \frac{Nm}{A}$$

$$\dot{\Psi} = 20 \frac{rad}{s}$$

$$T_x(t) = 2 Nm$$

$$b_m = 0.02 Nms/rad$$

At steady state, no damping (ideal)

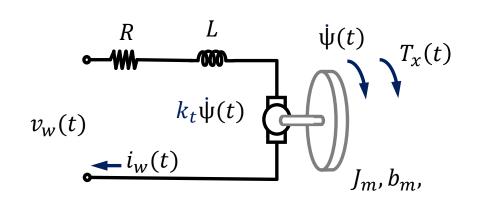
$$k_t i_w(t) = T_x(t) \to i_w(t) = \frac{T_x(t)}{k_t}$$
$$v_w(t) = i_w(t)R + k_t \dot{\psi}(t)$$

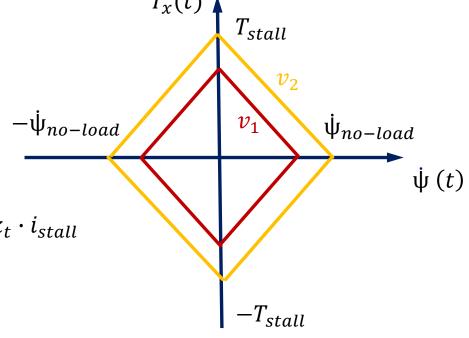
$$v_w(t) = \frac{RT_x(t)}{k_t} + k_t \dot{\Psi}(t)$$

$$T_{x}(t) = \frac{k_t^2}{R} \dot{\psi}(t) + \frac{k_t}{R} v_w(t)$$

Slope of diamond

Stall torque  $\rightarrow$   $T_{stall} = \frac{k_t}{R} v_w = k_t \cdot i_{stall}$ No-load speed  $\rightarrow$   $\dot{\theta}_{no-load} = \frac{v_w}{k_t}$ 





Mech.

Elec.

Power:  $T_x(t)\dot{\theta}_1(t)$   $v_w(t)i_w(t)$ 

- Mechanical and electrical power:
- Power is the derivative of energy
- Positive electrical power:

$$v_w(t) = i_w(t)R + L \frac{d}{dt}i_w(t) + k_t \dot{\Psi}(t)$$

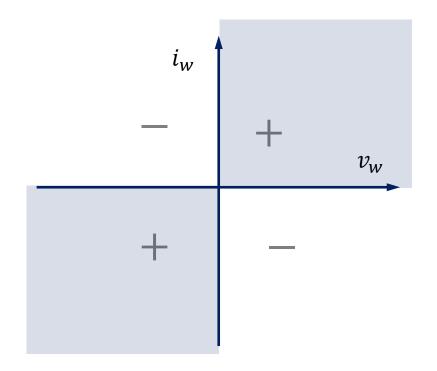
$$+ + + /- +$$

$$v_w(t) = i_w(t)R + L \frac{d}{dt}i_w(t) + k_t \dot{\psi}(t)$$

$$- +/- -$$

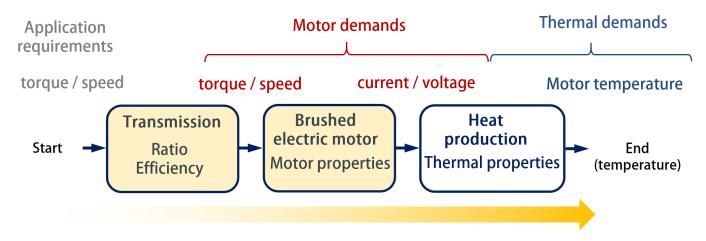
Negative power: torque and velocity in opposite direction

Positive power: torque and velocity in same direction



Positive power: 'motoring'

Negative power: 'generating'



- So far, we have learned how to model a transmission and motor
- Next, we will examine how heat affects component selection
- But before that, we will go over a simple physics-based model of a ball-bot
- We will focus on Newton's second law:

$$Force = mass \cdot acceleration \qquad F = m \cdot \ddot{x}$$
 
$$Torque = Moment \ of \ inertia \cdot angular \ acceleration \qquad T = J \cdot \ddot{\theta}$$

To do this, we need an understanding of the forces and torques

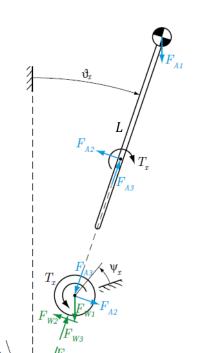
# Mechanical Modeling of a Planar Ball-Bot

$$F_{K1} = g \cdot m_k$$

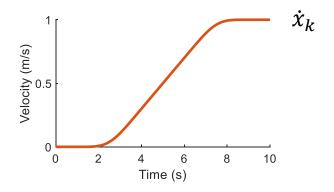
$$F_{W1} = g \cdot m_W$$

$$F_{A1} = g \cdot m_a$$

Ball radius  $r_k$ Wheel radius  $r_w$ 



If we want to predict the torque required, we need to generate a motion profile.



$$\dot{\varphi} = \frac{\dot{x}_k}{r_k}$$

Intermediate var: 
$$\gamma = L \cdot m_a + (r_k + r_w) \cdot m_w$$

**Contact force:** 

$$F_{W2} = (m_a + m_w) \cdot (g \cdot \sin(\theta) - r_k \ddot{\varphi} \cos(\theta)) - \gamma \ddot{\theta}$$