

6/6/12

Errata Sheet

PPI Handbook of Polyethylene Pipe, 2nd ed.

Chapter 6, Design of PE Piping Systems

Page 217

The equation for calculating $\Delta X/D_M$ uses the apparent modulus for the condition of a rapidly increasing stress or strain to solve for deflection due to only dead load; this calculation uses $E = 130,000$ psi per as shown in Table B2.1, p. 101. Instead, use E from Chp. 3, p. 99, Table B.1.1; for example, for PE 4710 and a load duration of 100 years, $E = 28,000$ psi. Also, under SOLUTION above the equation for $\Delta X/DM$, change '130,000' to "28,000" and change 'B2.1' to "B1.1".

Page 229

Under '(From Equation 3-23)' change "0.57" to 0.56" to match the answer shown above in the equation for VAF

Page 236, Table 3-14

Change the heading of the 2nd column from "Saturated, unit weight of ground water, pcf" to "Saturated Unit Weight of Soil, pcf". Keep the current symbol w_s as shown.

Change the heading in the 3rd column from "Dry, the weight of saturated soil above the pipe, lbs per foot of pipe" to 'Dry Unit Weight of Soil, pcf'. Keep the shown symbol w_d as shown.

Page 261, Appendix A.3

The definition of the variable C (Hazen-Williams Friction Factor ...) incorrectly refers the reader to see table 1-7. Change 'table 1-7' to "definitions below equation 2-12, p. 175"

Chapter 7, Underground Installation of PE Piping

p. 292, Table 4

Change title of Table to 'Minimum Long-Term Bend Radius for PE Pipe Installed in Open Cut Trench'

Change the heading of 1st column to "Dimension Ratio DR"

Chapter 8, Above-Ground Applications for PE Pipe

Page 318 Equation 6 is applicable where support spacing is relatively large and the spacing and support rigidity do not induce axial compression in the pipeline in response to changes in temperature.

Page 320 Equation 8 is applicable only when thermal expansion of the supported pipeline is not anticipated.

Page 322/323 Equation 11 and 13 are applicable where support spacing and support rigidity do not induce axial compression in the pipeline in response to changes in temperature.

Note: Where spans between support spacing are relatively short and the span length and support rigidity induce axial compression in the pipeline in response to changes in temperature, the reader is referred to 'Roark's Formulas for Stress & Strain' by Warren C. Young, Table 10, Page 166.

Chapter 10, Marine Installations

P. 379 Change title of table to: Table 3 ~~Pipe Diameter Multipliers for the Determining of Minimum~~ Short-Term Bending Radius'

Page 363, Step 3: Add this new Paragraph under the heading of Step 3: General: Calculation of 'maximum weighting' (Step 3b) and calculations to determine the percent air entrapment that may cause the pipe to refloat (Step 3d) are based on the selected DR. Calculations to determine the recommended weighting (Step 3c) is based on achieving a weighting that is 'equivalent' to the weight of the water displaced by the pipe and is independent of DR. Other marine installation handbooks include calculations to determine the recommended weighting that are dependent on DR. Contact the pipe manufacturer or contractors experienced in this field for additional advice.

Page 363, Step 3a: Change the title of this section to "Buoyant Force as a Percentage of Air Inside the Pipe."

Page 364, Step 3a: Add the following text to the end of Step 3a: "In a pond or otherwise under conditions of no current, the minimum weight required to anchor the pipe equals the buoyant force times a safety factor. The weight of individual anchors may be found by multiplying the safety factored buoyant force (per unit length) times the anchor spacing."

Page 365, Step 3b: Change the title of this section to "Practical Limit of Individual Ballast Weight (in air) for Towing of 100% Air Filled Pipe"

Page 365, Step 3c: Add the following as the first sentence in this section: The methodology described in this section is for pipelines that remain full of water, such as pressurized water lines. If you put in a line that contains primarily air, the weighting factors will not be conservative and may not be sufficient to prevent flotation.

Page 367, step 3d: The ratio of the selected ballast weight (Step 3c) divided by the minimum ballast weight to offset anticipated % air (Step 3a) may be regarded as the Factor of Safety against refloating the pipe. This may provide some safety factor against air entrapment but at the cost of reducing the safety factor against current moving the line.

Page 392-393; Table A-3-1 and Table A-3-2 Add this text before Table A-3-1:

The ballast physical dimensions, ballast weights, and ballast spacing shown in tables A-3-1 and A-3-2 are approximate and are often significantly different than when calculated from first principles.

The reader is advised to calculate the volumes of anchor blocks from first principles and to determine weights based on the densities for 'plain' and 'reinforced' concrete shown on page 369. As there is often a discrepancy between the 'produced' weight of the anchor block versus the theoretical weight, the reader is referred to Section 3e for advice about adjusting the ballast spacing to achieve the desired 'weighting percentage'.

The Tables assume the pipelines remain full of water, such as pressurized water lines. If you put in a line that contains primarily air, the weighting factors will not be conservative and may not be sufficient to prevent flotation.

Chapter 12, Horizontal Directional Drilling

P. 435

TABLE 1

Safe Pull Tensile Stress @ 73°F

	Proposed Table 1					
	Typical Safe Pull Stress (psi) @ 73°F			Time under tension design factor per ASTM F1804, note 4		
	PE 2xxx	PE 3608	PE 4710			
	ASTM D3350 Cell Classification	PE234373	PE345464		PE445574	
Minimum Tensile yield strength (psi) per ASTM D3350	2600	3000	3500			
Tensile yield design factor per ASTM F1804		0.4	0.4	0.4		
Duration (hours)						
0.5		1040	1200	1400		1.0
		(use 1050)				
1		1040	1200	1400		1.0
		(use 1050)				
12		988	1140	1330		0.95
(also used in PPI BoreAid and Calculator)		(use 1000)	(use 1150)	(use 1300)		
24		946	1092	1274	0.91	
		(use 950)	(use 1100)	(use 1250)		

P. 423, Pilot Hole Reaming add the text in red: Normal-oversizing may be from 1.2 to 1.5 times the diameter of the carrier pipe, **but at least 4" larger than the diameter of the carrier pipe.**

Index The page numbers that are being referred to in the Index do not match the text. Reader is encouraged to use the search engine on the PPI website at <http://plasticpipe.org/search.html>