

## Procedure for Analysis

Using potential-energy methods, the equilibrium positions and the stability of a body or a system of connected bodies having a single degree of freedom can be obtained by applying the following procedure.

### Potential Function.

- Sketch the system so that it is in the *arbitrary position* specified by the coordinate  $q$ .
- Establish a horizontal *datum* through a *fixed point*\* and express the gravitational potential energy  $V_g$  in terms of the weight  $W$  of each member and its vertical distance  $y$  from the datum,  $V_g = Wy$ .
- Express the elastic potential energy  $V_e$  of the system in terms of the stretch or compression,  $s$ , of any connecting spring,  $V_e = \frac{1}{2}ks^2$ .
- Formulate the potential function  $V = V_g + V_e$  and express the *position coordinates*  $y$  and  $s$  in terms of the single coordinate  $q$ .

### Equilibrium Position.

- The equilibrium position of the system is determined by taking the first derivative of  $V$  and setting it equal to zero,  $dV/dq = 0$ .

### Stability.

- Stability at the equilibrium position is determined by evaluating the second or higher-order derivatives of  $V$ .
- If the second derivative is greater than zero, the system is stable; if all derivatives are equal to zero, the system is in neutral equilibrium; and if the second derivative is less than zero, the system is unstable.

\*The location of the datum is *arbitrary*, since only the *changes* or differentials of  $V$  are required for investigation of the equilibrium position and its stability.

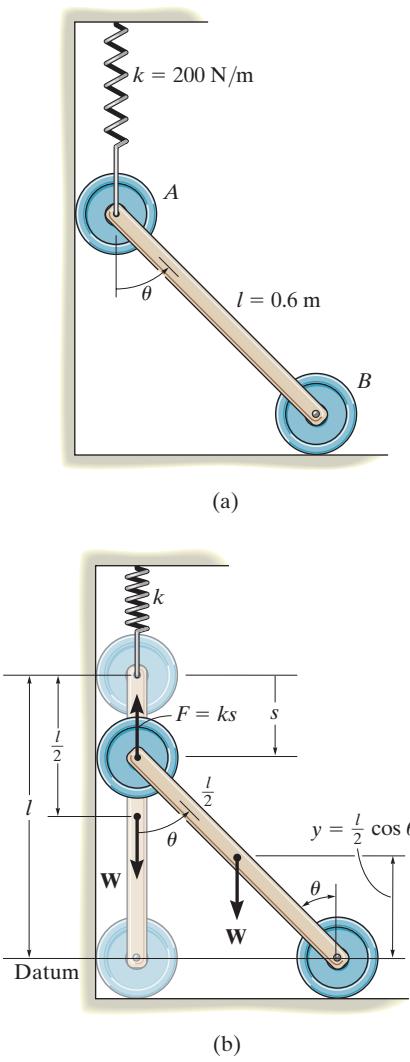


Fig. 11-17

The uniform link shown in Fig. 11-17a has a mass of 10 kg. If the spring is unstretched when  $\theta = 0^\circ$ , determine the angle  $\theta$  for equilibrium and investigate the stability at the equilibrium position.

### SOLUTION

**Potential Function.** The datum is established at the bottom of the link, Fig. 11-17b. When the link is located in the arbitrary position  $\theta$ , the spring increases its potential energy by stretching and the weight decreases its potential energy. Hence,

$$V = V_e + V_g = \frac{1}{2}ks^2 + Wy$$

Since  $l = s + l \cos \theta$  or  $s = l(1 - \cos \theta)$ , and  $y = (l/2) \cos \theta$ , then

$$V = \frac{1}{2}kl^2(1 - \cos \theta)^2 + W\left(\frac{l}{2} \cos \theta\right)$$

**Equilibrium Position.** The first derivative of  $V$  is

$$\frac{dV}{d\theta} = kl^2(1 - \cos \theta) \sin \theta - \frac{Wl}{2} \sin \theta = 0$$

or

$$l\left[kl(1 - \cos \theta) - \frac{W}{2}\right] \sin \theta = 0$$

This equation is satisfied provided

$$\sin \theta = 0 \quad \theta = 0^\circ \quad \text{Ans.}$$

or

$$\theta = \cos^{-1}\left(1 - \frac{W}{2kl}\right) = \cos^{-1}\left[1 - \frac{10(9.81)}{2(200)(0.6)}\right] = 53.8^\circ \quad \text{Ans.}$$

**Stability.** The second derivative of  $V$  is

$$\begin{aligned} \frac{d^2V}{d\theta^2} &= kl^2(1 - \cos \theta) \cos \theta + kl^2 \sin \theta \sin \theta - \frac{Wl}{2} \cos \theta \\ &= kl^2(\cos \theta - \cos 2\theta) - \frac{Wl}{2} \cos \theta \end{aligned}$$

Substituting values for the constants, with  $\theta = 0^\circ$  and  $\theta = 53.8^\circ$ , yields

$$\begin{aligned} \left.\frac{d^2V}{d\theta^2}\right|_{\theta=0^\circ} &= 200(0.6)^2(\cos 0^\circ - \cos 0^\circ) - \frac{10(9.81)(0.6)}{2} \cos 0^\circ \\ &= -29.4 < 0 \quad (\text{unstable equilibrium at } \theta = 0^\circ) \quad \text{Ans.} \end{aligned}$$

$$\begin{aligned} \left.\frac{d^2V}{d\theta^2}\right|_{\theta=53.8^\circ} &= 200(0.6)^2(\cos 53.8^\circ - \cos 107.6^\circ) - \frac{10(9.81)(0.6)}{2} \cos 53.8^\circ \\ &= 46.9 > 0 \quad (\text{stable equilibrium at } \theta = 53.8^\circ) \quad \text{Ans.} \end{aligned}$$

**EXAMPLE | 11.6**

If the spring  $AD$  in Fig. 11–18a has a stiffness of  $18 \text{ kN/m}$  and is unstretched when  $\theta = 60^\circ$ , determine the angle  $\theta$  for equilibrium. The load has a mass of  $1.5 \text{ Mg}$ . Investigate the stability at the equilibrium position.

**SOLUTION**

**Potential Energy.** The gravitational potential energy for the load with respect to the fixed datum, shown in Fig. 11–18b, is

$V_g = mgy = 1500(9.81) \text{ N}[(4 \text{ m}) \sin \theta + h] = 58860 \sin \theta + 14715h$   
where  $h$  is a constant distance. From the geometry of the system, the elongation of the spring when the load is on the platform is  $s = (4 \text{ m}) \cos \theta - (4 \text{ m}) \cos 60^\circ = (4 \text{ m}) \cos \theta - 2 \text{ m}$ .

Thus, the elastic potential energy of the system is

$$V_e = \frac{1}{2}ks^2 = \frac{1}{2}(18000 \text{ N/m})(4 \text{ m} \cos \theta - 2 \text{ m})^2 = 9000(4 \cos \theta - 2)^2$$

The potential energy function for the system is therefore

$$V = V_g + V_e = 58860 \sin \theta + 14715h + 9000(4 \cos \theta - 2)^2 \quad (1)$$

**Equilibrium.** When the system is in equilibrium,

$$\begin{aligned} \frac{dV}{d\theta} &= 58860 \cos \theta + 18000(4 \cos \theta - 2)(-4 \sin \theta) = 0 \\ 58860 \cos \theta - 288000 \sin \theta \cos \theta + 144000 \sin \theta &= 0 \end{aligned}$$

Since  $\sin 2\theta = 2 \sin \theta \cos \theta$ ,

$$58860 \cos \theta - 144000 \sin 2\theta + 144000 \sin \theta = 0$$

Solving by trial and error,

$$\theta = 28.18^\circ \text{ and } \theta = 45.51^\circ$$

**Stability.** Taking the second derivative of Eq. 1,

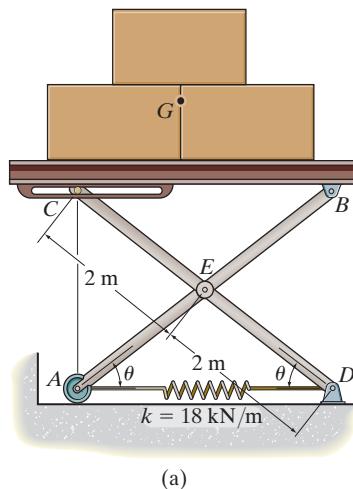
$$\frac{d^2V}{d\theta^2} = -58860 \sin \theta - 288000 \cos 2\theta + 144000 \cos \theta$$

Substituting  $\theta = 28.18^\circ$  yields

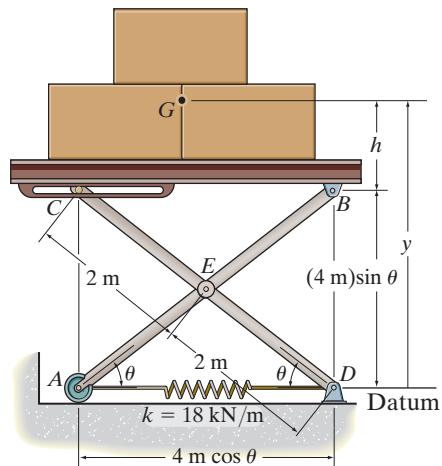
$$\frac{d^2V}{d\theta^2} = -60402 < 0 \quad \text{Unstable}$$

And for  $\theta = 45.51^\circ$ ,

$$\frac{d^2V}{d\theta^2} = 64073 > 0 \quad \text{Stable} \quad \text{Ans.}$$

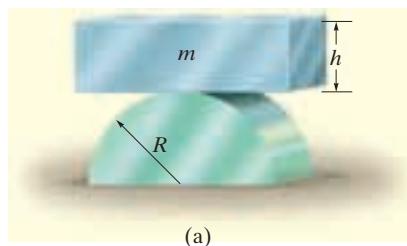


(a)



(b)

**Fig. 11–18**



(a)

The uniform block having a mass  $m$  rests on the top surface of the half cylinder, Fig. 11-19a. Show that this is a condition of unstable equilibrium if  $h > 2R$ .

### SOLUTION

**Potential Function.** The datum is established at the base of the cylinder, Fig. 11-19b. If the block is displaced by an amount  $\theta$  from the equilibrium position, the potential function is

$$\begin{aligned}V &= V_e + V_g \\&= 0 + mg y\end{aligned}$$

From Fig. 11-19b,

$$y = \left( R + \frac{h}{2} \right) \cos \theta + R\theta \sin \theta$$

Thus,

$$V = mg \left[ \left( R + \frac{h}{2} \right) \cos \theta + R\theta \sin \theta \right]$$

### Equilibrium Position.

$$\begin{aligned}\frac{dV}{d\theta} &= mg \left[ -\left( R + \frac{h}{2} \right) \sin \theta + R \sin \theta + R\theta \cos \theta \right] = 0 \\&= mg \left( -\frac{h}{2} \sin \theta + R\theta \cos \theta \right) = 0\end{aligned}$$

Note that  $\theta = 0^\circ$  satisfies this equation.

**Stability.** Taking the second derivative of  $V$  yields

$$\frac{d^2V}{d\theta^2} = mg \left( -\frac{h}{2} \cos \theta + R \cos \theta - R\theta \sin \theta \right)$$

At  $\theta = 0^\circ$ ,

$$\left. \frac{d^2V}{d\theta^2} \right|_{\theta=0^\circ} = -mg \left( \frac{h}{2} - R \right)$$

Since all the constants are positive, the block is in unstable equilibrium provided  $h > 2R$ , because then  $d^2V/d\theta^2 < 0$ .

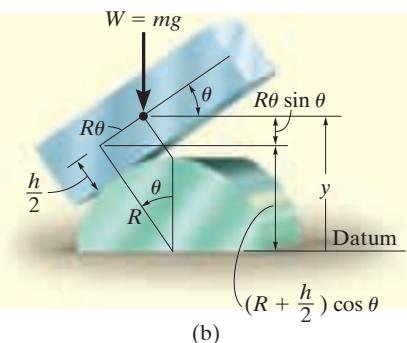


Fig. 11-19

**11-26.** The potential energy of a one-degree-of-freedom system is defined by  $V = (20x^3 - 10x^2 - 25x - 10)$  ft·lb, where  $x$  is in ft. Determine the equilibrium positions and investigate the stability for each position.

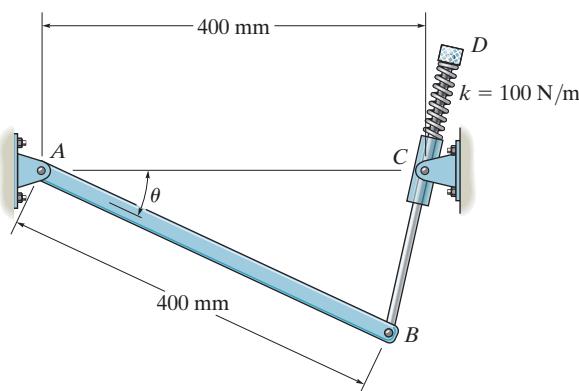
**11-27.** If the potential function for a conservative one-degree-of-freedom system is  $V = (12 \sin 2\theta + 15 \cos \theta)$  J, where  $0^\circ < \theta < 180^\circ$ , determine the positions for equilibrium and investigate the stability at each of these positions.

**\*11-28.** If the potential function for a conservative one-degree-of-freedom system is  $V = (8x^3 - 2x^2 - 10)$  J, where  $x$  is given in meters, determine the positions for equilibrium and investigate the stability at each of these positions.

**11-29.** If the potential function for a conservative one-degree-of-freedom system is  $V = (10 \cos 2\theta + 25 \sin \theta)$  J, where  $0^\circ < \theta < 180^\circ$ , determine the positions for equilibrium and investigate the stability at each of these positions.

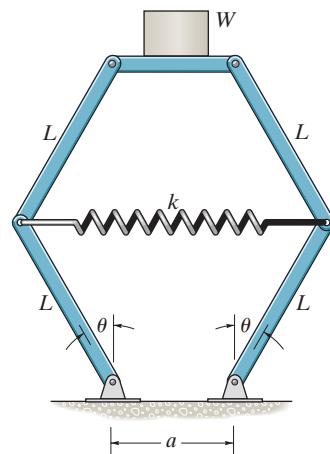
**11-30.** If the potential energy for a conservative one-degree-of-freedom system is expressed by the relation  $V = (4x^3 - x^2 - 3x + 10)$  ft·lb, where  $x$  is given in feet, determine the equilibrium positions and investigate the stability at each position.

**11-31.** The uniform link  $AB$ , has a mass of 3 kg and is pin connected at both of its ends. The rod  $BD$ , having negligible weight, passes through a swivel block at  $C$ . If the spring has a stiffness of  $k = 100 \text{ N/m}$  and is unstretched when  $\theta = 0^\circ$ , determine the angle  $\theta$  for equilibrium and investigate the stability at the equilibrium position. Neglect the size of the swivel block.



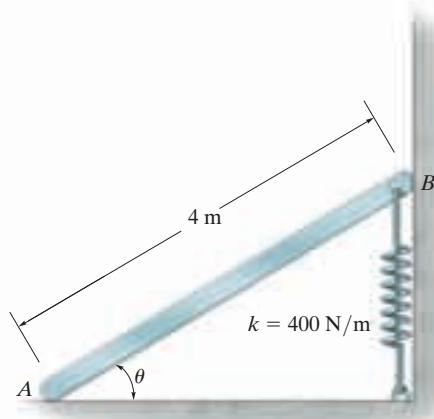
Prob. 11-31

**\*11-32.** The spring of the scale has an unstretched length of  $a$ . Determine the angle  $\theta$  for equilibrium when a weight  $W$  is supported on the platform. Neglect the weight of the members. What value  $W$  would be required to keep the scale in neutral equilibrium when  $\theta = 0^\circ$ ?



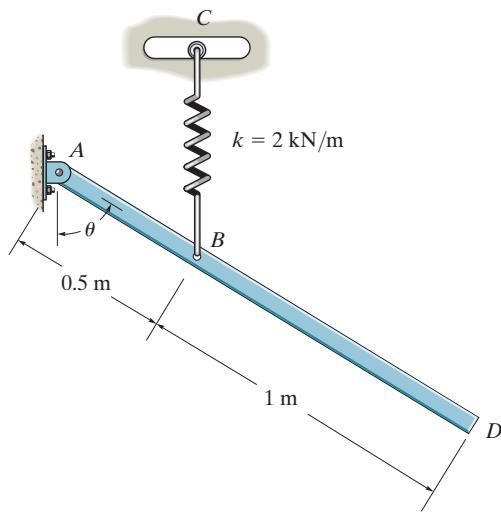
Prob. 11-32

**11-33.** The uniform bar has a mass of 80 kg. Determine the angle  $\theta$  for equilibrium and investigate the stability of the bar when it is in this position. The spring has an unstretched length when  $\theta = 90^\circ$ .



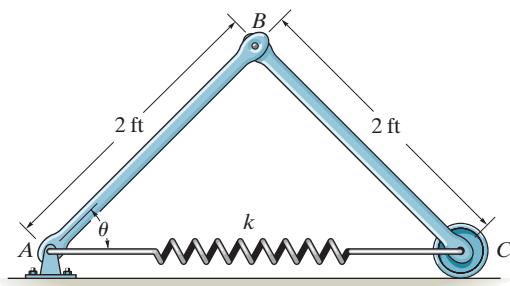
Prob. 11-33

- 11-34.** The uniform bar  $AD$  has a mass of 20 kg. If the attached spring is unstretched when  $\theta = 90^\circ$ , determine the angle  $\theta$  for equilibrium. Note that the spring always remains in the vertical position due to the roller guide. Investigate the stability of the bar when it is in the equilibrium position.



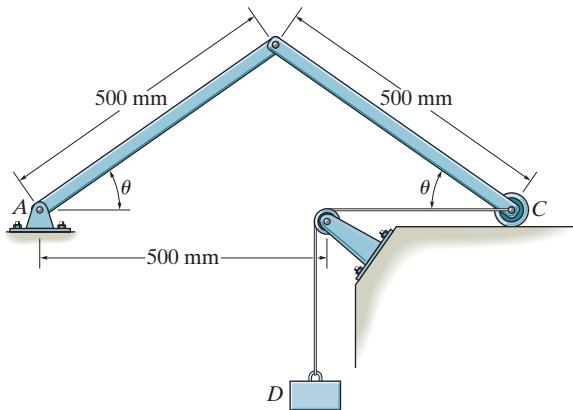
Prob. 11-34

- 11-35.** The two bars each have a weight of 8 lb. Determine the required stiffness  $k$  of the spring so that the two bars are in equilibrium when  $\theta = 30^\circ$ . The spring has an unstretched length of 1 ft.



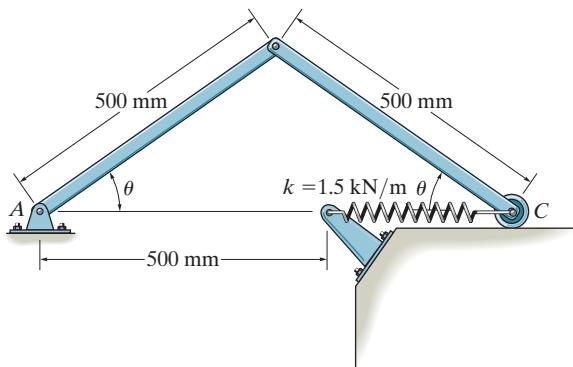
Prob. 11-35

- \*11-36.** Determine the angle  $\theta$  for equilibrium and investigate the stability at this position. The bars each have a mass of 3 kg and the suspended block  $D$  has a mass of 7 kg. Cord  $DC$  has a total length of 1 m.



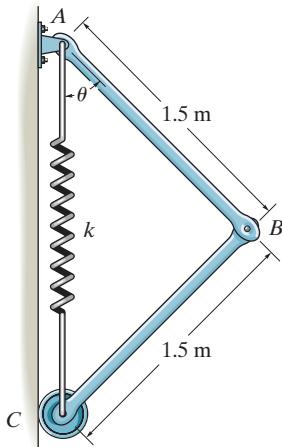
Prob. 11-36

- 11-37.** Determine the angle  $\theta$  for equilibrium and investigate the stability at this position. The bars each have a mass of 10 kg and the spring has an unstretched length of 100 mm.



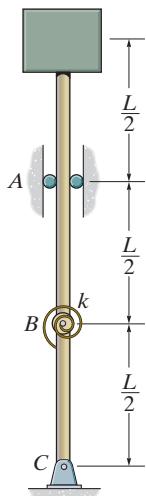
Prob. 11-37

**11–38.** The two bars each have a mass of 8 kg. Determine the required stiffness  $k$  of the spring so that the two bars are in equilibrium when  $\theta = 60^\circ$ . The spring has an unstretched length of 1 m. Investigate the stability of the system at the equilibrium position.



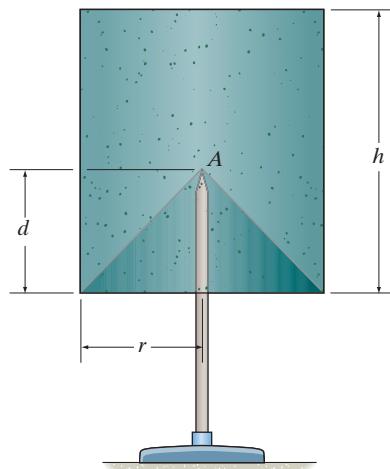
Prob. 11-38

**11–39.** A spring with a torsional stiffness  $k$  is attached to the hinge at  $B$ . It is unstretched when the rod assembly is in the vertical position. Determine the weight  $W$  of the block that results in neutral equilibrium. Hint: Establish the potential energy function for a small angle  $\theta$ , i.e., approximate  $\sin \theta \approx 0$ , and  $\cos \theta \approx 1 - \theta^2/2$ .



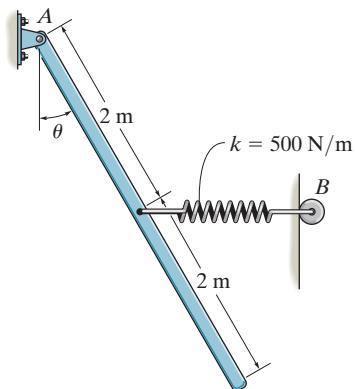
Prob. 11-39

**\*11–40.** A conical hole is drilled into the bottom of the cylinder, which is supported on the fulcrum at  $A$ . Determine the minimum distance  $d$  in order for it to remain in stable equilibrium.



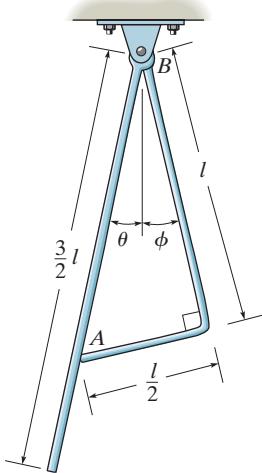
Prob. 11-40

**11–41.** The uniform rod has a mass of 100 kg. If the spring is unstretched when  $\theta = 60^\circ$ , determine the angle  $\theta$  for equilibrium and investigate the stability at the equilibrium position. The spring is always in the horizontal position due to the roller guide at  $B$ .



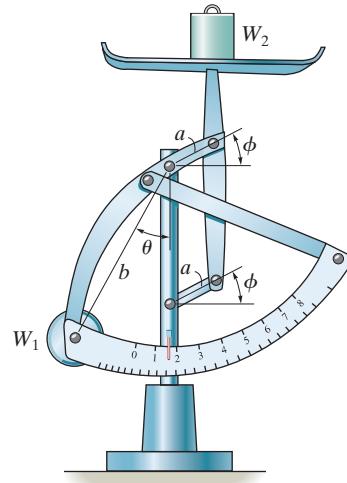
Prob. 11-41

- 11-42.** Each bar has a mass per length of  $m_0$ . Determine the angles  $\theta$  and  $\phi$  at which they are suspended in equilibrium. The contact at  $A$  is smooth, and both are pin connected at  $B$ .



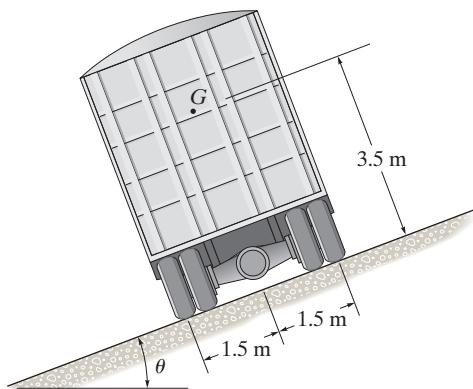
Prob. 11-42

- \*11-44.** The small postal scale consists of a counterweight  $W_1$ , connected to the members having negligible weight. Determine the weight  $W_2$  that is on the pan in terms of the angles  $\theta$  and  $\phi$  and the dimensions shown. All members are pin connected.



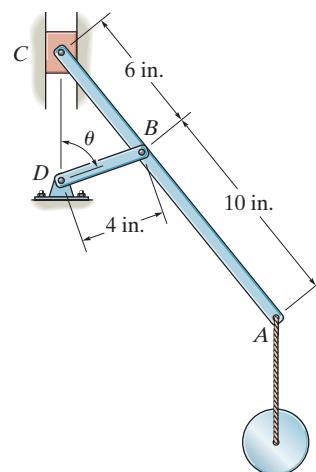
Prob. 11-44

- 11-43.** The truck has a mass of 20 Mg and a mass center at  $G$ . Determine the steepest grade  $\theta$  along which it can park without overturning and investigate the stability in this position.



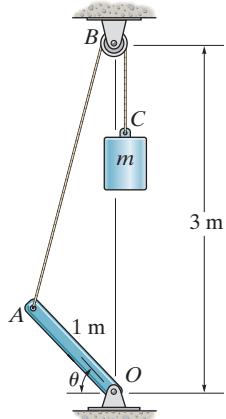
Prob. 11-43

- 11-45.** A 3-lb weight is attached to the end of rod  $ABC$ . If the rod is supported by a smooth slider block at  $C$  and rod  $BD$ , determine the angle  $\theta$  for equilibrium. Neglect the weight of the rods and the slider.

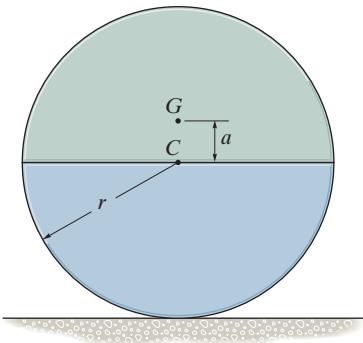


Prob. 11-45

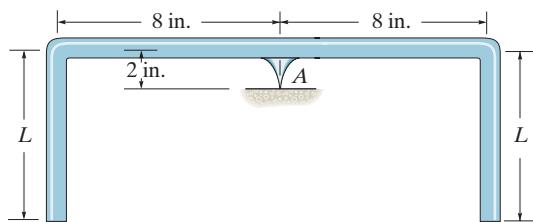
- 11-46.** If the uniform rod  $OA$  has a mass of 12 kg, determine the mass  $m$  that will hold the rod in equilibrium when  $\theta = 30^\circ$ . Point  $C$  is coincident with  $B$  when  $OA$  is horizontal. Neglect the size of the pulley at  $B$ .

**Prob. 11-46**

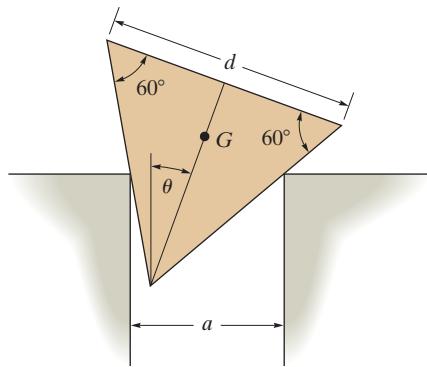
- 11-47.** The cylinder is made of two materials such that it has a mass of  $m$  and a center of gravity at point  $G$ . Show that when  $G$  lies above the centroid  $C$  of the cylinder, the equilibrium is unstable.

**Prob. 11-47**

- \*11-48.** The bent rod has a weight of 5 lb/ft. A pivot is attached at its center  $A$  and the rod is balanced as shown. Determine the length  $L$  of its vertical segments so that it remains in neutral equilibrium. Neglect the thickness of the rod.

**Prob. 11-48**

- 11-49.** The triangular block of weight  $W$  rests on the smooth corners which are a distance  $a$  apart. If the block has three equal sides of length  $d$ , determine the angle  $\theta$  for equilibrium.

**Prob. 11-49**

## CHAPTER REVIEW

### Principle of Virtual Work

The forces on a body will do *virtual work* when the body undergoes an *imaginary* differential displacement or rotation.

For equilibrium, the sum of the virtual work done by all the forces acting on the body must be equal to zero for any virtual displacement. This is referred to as the *principle of virtual work*, and it is useful for finding the equilibrium configuration for a mechanism or a reactive force acting on a series of connected members.

If the system of connected members has one degree of freedom, then its position can be specified by one independent coordinate, such as  $\theta$ .

To apply the principle of virtual work, it is first necessary to use *position coordinates* to locate all the forces and moments on the mechanism that will do work when the mechanism undergoes a virtual movement  $\delta\theta$ .

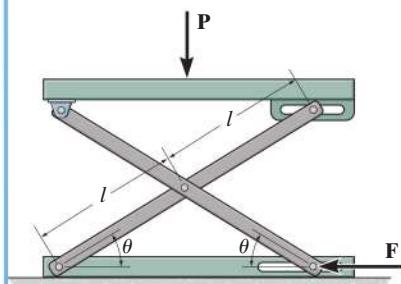
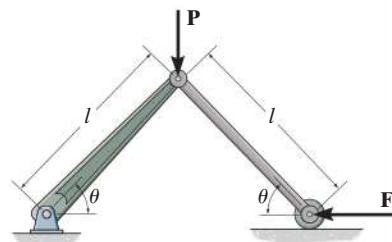
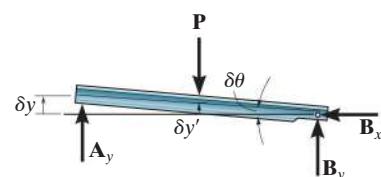
The coordinates are related to the independent coordinate  $\theta$  and then these expressions are differentiated in order to relate the *virtual* coordinate displacements to the virtual displacement  $\delta\theta$ .

Finally, the equation of virtual work is written for the mechanism in terms of the common virtual displacement  $\delta\theta$ , and then it is set equal to zero. By factoring  $\delta\theta$  out of the equation, it is then possible to determine either the unknown force or couple moment, or the equilibrium position  $\theta$ .

$\delta y, \delta y'$ —virtual displacements

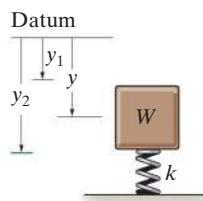
$\delta\theta$ —virtual rotation

$$\delta U = 0$$



### Potential-Energy Criterion for Equilibrium

When a system is subjected only to conservative forces, such as weight and spring forces, then the equilibrium configuration can be determined using the *potential-energy function*  $V$  for the system.



$$V = V_g + V_e = -Wy + \frac{1}{2}ky^2$$

The potential-energy function is established by expressing the weight and spring potential energy for the system in terms of the independent coordinate  $q$ .

Once the potential-energy function is formulated, its first derivative is set equal to zero. The solution yields the equilibrium position  $q_{\text{eq}}$  for the system.

$$\frac{dV}{dq} = 0$$

The stability of the system can be investigated by taking the second derivative of  $V$ .

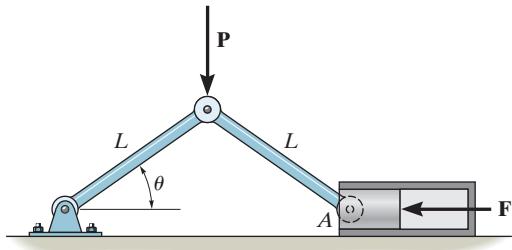
$$\frac{dV}{dq} = 0, \quad \frac{d^2V}{dq^2} > 0 \quad \text{stable equilibrium}$$

$$\frac{dV}{dq} = 0, \quad \frac{d^2V}{dq^2} < 0 \quad \text{unstable equilibrium}$$

$$\frac{dV}{dq} = \frac{d^2V}{dq^2} = \frac{d^3V}{dq^3} = \dots = 0 \quad \text{neutral equilibrium}$$

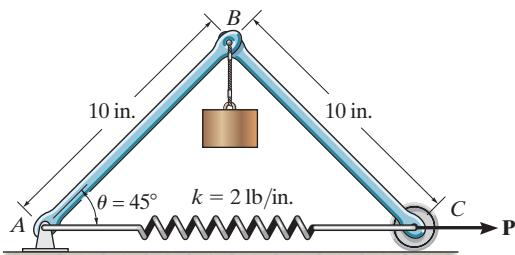
## REVIEW PROBLEMS

**R11-1.** The toggle joint is subjected to the load  $P$ . Determine the compressive force  $F$  it creates on the cylinder at  $A$  as a function of  $\theta$ .



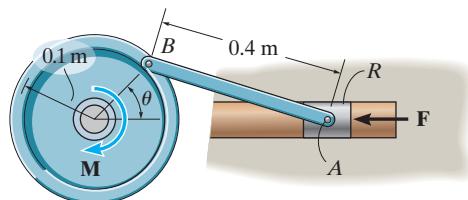
Prob. R11-1

**R11-2.** The uniform links  $AB$  and  $BC$  each weigh 2 lb and the cylinder weighs 20 lb. Determine the horizontal force  $\mathbf{P}$  required to hold the mechanism in the position when  $\theta = 45^\circ$ . The spring has an unstretched length of 6 in.



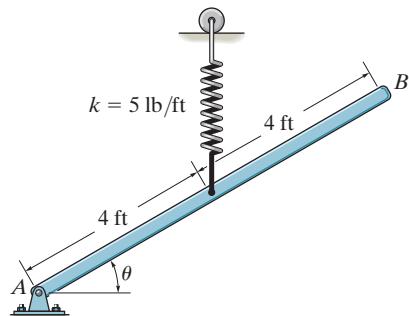
Prob. R11-2

**R11-3.** The punch press consists of the ram  $R$ , connecting rod  $AB$ , and a flywheel. If a torque of  $M = 50 \text{ N}\cdot\text{m}$  is applied to the flywheel, determine the force  $\mathbf{F}$  applied at the ram to hold the rod in the position  $\theta = 60^\circ$ .



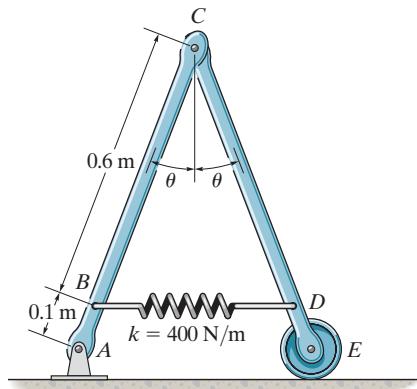
Prob. R11-3

**R11-4.** The uniform bar  $AB$  weighs 10 lb. If the attached spring is unstretched when  $\theta = 90^\circ$ , use the method of virtual work and determine the angle  $\theta$  for equilibrium. Note that the spring always remains in the vertical position due to the roller guide.



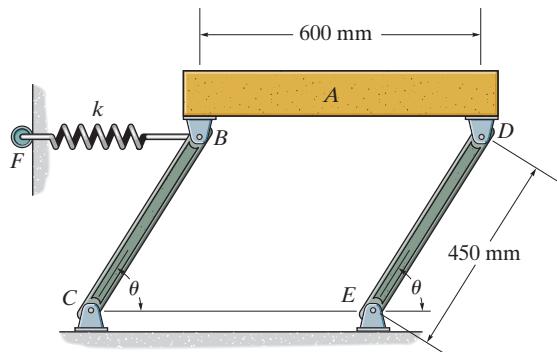
Prob. R11-4

**R11-5.** The spring has an unstretched length of 0.3 m. Determine the angle  $\theta$  for equilibrium if the uniform links each have a mass of 5 kg.



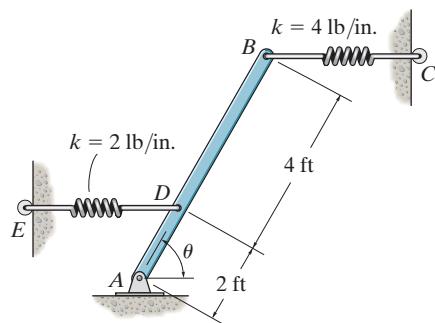
Prob. R11-5

**R11-6.** Determine the angle  $\theta$  for equilibrium and investigate the stability of the mechanism in this position. The spring has a stiffness of  $k = 1.5 \text{ kN/m}$  and is unstretched when  $\theta = 90^\circ$ . The block A has a mass of 40 kg. Neglect the mass of the links.



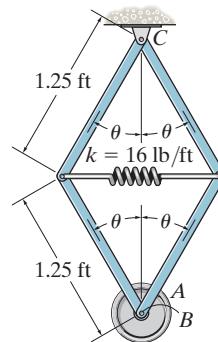
Prob. R11-6

**R11-7.** The uniform bar  $AB$  weighs 100 lb. If both springs  $DE$  and  $BC$  are unstretched when  $\theta = 90^\circ$ , determine the angle  $\theta$  for equilibrium using the principle of potential energy. Investigate the stability at the equilibrium position. Both springs always act in the horizontal position because of the roller guides at  $C$  and  $E$ .



Prob. R11-7

**R11-8.** The spring attached to the mechanism has an unstretched length when  $\theta = 90^\circ$ . Determine the position  $\theta$  for equilibrium and investigate the stability of the mechanism at this position. Disk A is pin connected to the frame at B and has a weight of 20 lb. Neglect the weight of the bars.



Prob. R11-8

# A

# Mathematical Review and Expressions

## Geometry and Trigonometry Review

The angles  $\theta$  in Fig. A-1 are equal between the transverse and two parallel lines.

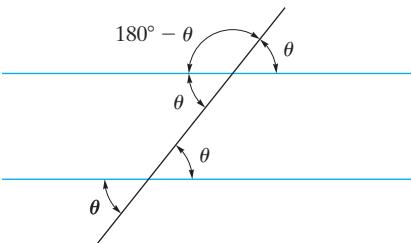


Fig. A-1

For a line and its normal, the angles  $\theta$  in Fig. A-2 are equal.

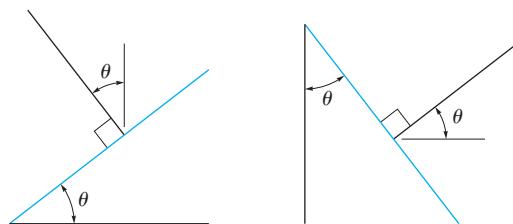


Fig. A-2

For the circle in Fig. A-3,  $s = \theta r$ , so that when  $\theta = 360^\circ = 2\pi$  rad then the circumference is  $s = 2\pi r$ . Also, since  $180^\circ = \pi$  rad, then  $\theta(\text{rad}) = (\pi/180^\circ)\theta^\circ$ . The area of the circle is  $A = \pi r^2$ .

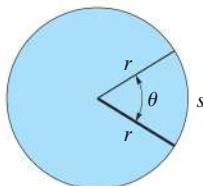


Fig. A-3

The sides of a similar triangle can be obtained by proportion as in Fig. A-4, where  $\frac{a}{A} = \frac{b}{B} = \frac{c}{C}$ .

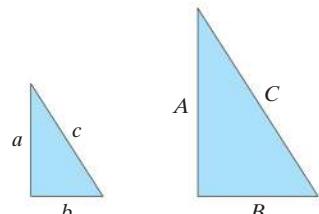


Fig. A-4

For the right triangle in Fig. A-5, the Pythagorean theorem is

$$h = \sqrt{(o)^2 + (a)^2}$$

The trigonometric functions are

$$\sin \theta = \frac{o}{h}$$

$$\cos \theta = \frac{a}{h}$$

$$\tan \theta = \frac{o}{a}$$

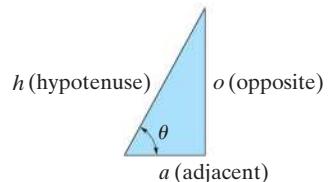


Fig. A-5

This is easily remembered as “soh, cah, toa,” i.e., the sine is the opposite over the hypotenuse, etc. The other trigonometric functions follow from this.

$$\csc \theta = \frac{1}{\sin \theta} = \frac{h}{o}$$

$$\sec \theta = \frac{1}{\cos \theta} = \frac{h}{a}$$

$$\cot \theta = \frac{1}{\tan \theta} = \frac{a}{o}$$

## Trigonometric Identities

$$\sin^2 \theta + \cos^2 \theta = 1$$

$$\sin(\theta \pm \phi) = \sin \theta \cos \phi \pm \cos \theta \sin \phi$$

$$\sin 2\theta = 2 \sin \theta \cos \theta$$

$$\cos(\theta \pm \phi) = \cos \theta \cos \phi \mp \sin \theta \sin \phi$$

$$\cos 2\theta = \cos^2 \theta - \sin^2 \theta$$

$$\cos \theta = \pm \sqrt{\frac{1 + \cos 2\theta}{2}}, \sin \theta = \pm \sqrt{\frac{1 - \cos 2\theta}{2}}$$

$$\tan \theta = \frac{\sin \theta}{\cos \theta}$$

$$1 + \tan^2 \theta = \sec^2 \theta \quad 1 + \cot^2 \theta = \csc^2 \theta$$

## Quadratic Formula

$$\text{If } ax^2 + bx + c = 0, \text{ then } x = \frac{-b \pm \sqrt{b^2 - 4ac}}{2a}$$

## Hyperbolic Functions

$$\sinh x = \frac{e^x - e^{-x}}{2},$$

$$\cosh x = \frac{e^x + e^{-x}}{2},$$

A

$$\tanh x = \frac{\sinh x}{\cosh x}$$

## Power-Series Expansions

$$\sin x = x - \frac{x^3}{3!} + \dots, \cos x = 1 - \frac{x^2}{2!} + \dots$$

$$\sinh x = x + \frac{x^3}{3!} + \dots, \cosh x = 1 + \frac{x^2}{2!} + \dots$$

## Derivatives

$$\frac{d}{dx}(u^n) = nu^{n-1} \frac{du}{dx} \quad \frac{d}{dx}(\sin u) = \cos u \frac{du}{dx}$$

$$\frac{d}{dx}(uv) = u \frac{dv}{dx} + v \frac{du}{dx} \quad \frac{d}{dx}(\cos u) = -\sin u \frac{du}{dx}$$

$$\frac{d}{dx}\left(\frac{u}{v}\right) = \frac{v \frac{du}{dx} - u \frac{dv}{dx}}{v^2} \quad \frac{d}{dx}(\tan u) = \sec^2 u \frac{du}{dx}$$

$$\frac{d}{dx}(\cot u) = -\csc^2 u \frac{du}{dx} \quad \frac{d}{dx}(\sinh u) = \cosh u \frac{du}{dx}$$

$$\frac{d}{dx}(\sec u) = \tan u \sec u \frac{du}{dx} \quad \frac{d}{dx}(\cosh u) = \sinh u \frac{du}{dx}$$

$$\frac{d}{dx}(\csc u) = -\csc u \cot u \frac{du}{dx}$$

## Integrals

$$\int x^n dx = \frac{x^{n+1}}{n+1} + C, n \neq -1$$

$$\int \frac{dx}{a+bx} = \frac{1}{b} \ln(a+bx) + C$$

$$\int \frac{dx}{a+bx^2} = \frac{1}{2\sqrt{-ab}} \ln \left[ \frac{a+x\sqrt{-ab}}{a-x\sqrt{-ab}} \right] + C, \\ ab < 0$$

$$\int \frac{x dx}{a+bx^2} = \frac{1}{2b} \ln(bx^2 + a) + C$$

$$\int \frac{x^2 dx}{a+bx^2} = \frac{x}{b} - \frac{a}{b\sqrt{ab}} \tan^{-1} \frac{x\sqrt{ab}}{a} + C, ab > 0$$

$$\int \sqrt{a+bx} dx = \frac{2}{3b} \sqrt{(a+bx)^3} + C$$

$$\int x \sqrt{a+bx} dx = \frac{-2(2a-3bx)\sqrt{(a+bx)^3}}{15b^2} + C$$

$$\int x^2 \sqrt{a+bx} dx = \frac{2(8a^2 - 12abx + 15b^2x^2)\sqrt{(a+bx)^3}}{105b^3} + C$$

$$\int \sqrt{a^2 - x^2} dx = \frac{1}{2} \left[ x \sqrt{a^2 - x^2} + a^2 \sin^{-1} \frac{x}{a} \right] + C, \\ a > 0$$

$$\int x \sqrt{a^2 - x^2} dx = -\frac{1}{3} \sqrt{(a^2 - x^2)^3} + C$$

$$\int x^2 \sqrt{a^2 - x^2} dx = -\frac{x}{4} \sqrt{(a^2 - x^2)^3}$$

$$+ \frac{a^2}{8} \left( x \sqrt{a^2 - x^2} + a^2 \sin^{-1} \frac{x}{a} \right) + C, a > 0$$

$$\int \sqrt{x^2 \pm a^2} dx =$$

$$\frac{1}{2} \left[ x \sqrt{x^2 \pm a^2} \pm a^2 \ln(x + \sqrt{x^2 \pm a^2}) \right] + C$$

$$\int x \sqrt{x^2 \pm a^2} dx = \frac{1}{3} \sqrt{(x^2 \pm a^2)^3} + C$$

$$\int x^2 \sqrt{x^2 \pm a^2} dx = \frac{x}{4} \sqrt{(x^2 \pm a^2)^3}$$

$$\mp \frac{a^2}{8} x \sqrt{x^2 \pm a^2} - \frac{a^4}{8} \ln(x + \sqrt{x^2 \pm a^2}) + C$$

$$\int \frac{dx}{\sqrt{a+bx}} = \frac{2\sqrt{a+bx}}{b} + C$$

$$\int \frac{x dx}{\sqrt{x^2 \pm a^2}} = \sqrt{x^2 \pm a^2} + C$$

$$\int \frac{dx}{\sqrt{a+bx+cx^2}} = \frac{1}{\sqrt{c}} \ln \left[ \sqrt{a+bx+cx^2} + x\sqrt{c} + \frac{b}{2\sqrt{c}} \right] + C, c > 0$$

$$= \frac{1}{\sqrt{-c}} \sin^{-1} \left( \frac{-2cx-b}{\sqrt{b^2-4ac}} \right) + C, c < 0$$

$$\int \sin x dx = -\cos x + C$$

$$\int \cos x dx = \sin x + C$$

$$\int x \cos(ax) dx = \frac{1}{a^2} \cos(ax) + \frac{x}{a} \sin(ax) + C$$

$$\int x^2 \cos(ax) dx = \frac{2x}{a^2} \cos(ax) + \frac{a^2x^2 - 2}{a^3} \sin(ax) + C$$

$$\int e^{ax} dx = \frac{1}{a} e^{ax} + C$$

$$\int x e^{ax} dx = \frac{e^{ax}}{a^2} (ax - 1) + C$$

$$\int \sinh x dx = \cosh x + C$$

$$\int \cosh x dx = \sinh x + C$$

# Fundamental Problems

## Partial Solutions And Answers

### Chapter 2

**F2-1.**

$$\begin{aligned} F_R &= \sqrt{(2 \text{ kN})^2 + (6 \text{ kN})^2 - 2(2 \text{ kN})(6 \text{ kN}) \cos 105^\circ} \\ &= 6.798 \text{ kN} = 6.80 \text{ kN} \\ \frac{\sin \phi}{6 \text{ kN}} &= \frac{\sin 105^\circ}{6.798 \text{ kN}}, \quad \phi = 58.49^\circ \\ \theta &= 45^\circ + \phi = 45^\circ + 58.49^\circ = 103^\circ \end{aligned}$$

*Ans.*

**F2-2.**  $F_R = \sqrt{200^2 + 500^2 - 2(200)(500) \cos 140^\circ}$

$$= 666 \text{ N}$$

*Ans.*

**F2-3.**  $F_R = \sqrt{600^2 + 800^2 - 2(600)(800) \cos 60^\circ}$

$$= 721.11 \text{ N} = 721 \text{ N}$$

$$\frac{\sin \alpha}{800} = \frac{\sin 60^\circ}{721.11}; \quad \alpha = 73.90^\circ$$

$$\phi = \alpha - 30^\circ = 73.90^\circ - 30^\circ = 43.9^\circ$$

*Ans.*

**F2-4.**  $\frac{F_u}{\sin 45^\circ} = \frac{30}{\sin 105^\circ}; \quad F_u = 22.0 \text{ lb}$

$$\frac{F_v}{\sin 30^\circ} = \frac{30}{\sin 105^\circ}; \quad F_v = 15.5 \text{ lb}$$

*Ans.*

*Ans.*

**F2-5.**  $\frac{F_{AB}}{\sin 105^\circ} = \frac{450}{\sin 30^\circ}$

$$F_{AB} = 869 \text{ lb}$$

$$\frac{F_{AC}}{\sin 45^\circ} = \frac{450}{\sin 30^\circ}$$

$$F_{AC} = 636 \text{ lb}$$

*Ans.*

*Ans.*

**F2-6.**  $\frac{F}{\sin 30^\circ} = \frac{6}{\sin 105^\circ} \quad F = 3.11 \text{ kN}$

$$\frac{F_v}{\sin 45^\circ} = \frac{6}{\sin 105^\circ} \quad F_v = 4.39 \text{ kN}$$

*Ans.*

*Ans.*

**F2-7.**  $(F_1)_x = 0 \quad (F_1)_y = 300 \text{ N}$

$$(F_2)_x = -(450 \text{ N}) \cos 45^\circ = -318 \text{ N}$$

$$(F_2)_y = (450 \text{ N}) \sin 45^\circ = 318 \text{ N}$$

$$(F_3)_x = (\frac{3}{5})600 \text{ N} = 360 \text{ N}$$

$$(F_3)_y = (\frac{4}{5})600 \text{ N} = 480 \text{ N}$$

*Ans.*

*Ans.*

*Ans.*

*Ans.*

**F2-8.**  $F_{Rx} = 300 + 400 \cos 30^\circ - 250(\frac{4}{5}) = 446.4 \text{ N}$

$$F_{Ry} = 400 \sin 30^\circ + 250(\frac{3}{5}) = 350 \text{ N}$$

$$F_R = \sqrt{(446.4)^2 + 350^2} = 567 \text{ N}$$

$$\theta = \tan^{-1} \frac{350}{446.4} = 38.1^\circ \angle \nwarrow$$

*Ans.*

*Ans.*

**F2-9.**

$$\begin{aligned} \stackrel{+}{\rightarrow}(F_R)_x &= \Sigma F_x; \\ (F_R)_x &= -(700 \text{ lb}) \cos 30^\circ + 0 + (\frac{3}{5})(600 \text{ lb}) \\ &= -246.22 \text{ lb} \\ \stackrel{+}{\uparrow}(F_R)_y &= \Sigma F_y; \\ (F_R)_y &= -(700 \text{ lb}) \sin 30^\circ - 400 \text{ lb} - (\frac{4}{5})(600 \text{ lb}) \\ &= -1230 \text{ lb} \\ F_R &= \sqrt{(246.22 \text{ lb})^2 + (1230 \text{ lb})^2} = 1254 \text{ lb} \quad \text{Ans.} \\ \phi &= \tan^{-1} \left( \frac{1230 \text{ lb}}{246.22 \text{ lb}} \right) = 78.68^\circ \\ \theta &= 180^\circ + \phi = 180^\circ + 78.68^\circ = 259^\circ \quad \text{Ans.} \end{aligned}$$

**F2-10.**  $\stackrel{+}{\rightarrow}(F_R)_x = \Sigma F_x;$

$$\begin{aligned} 750 \text{ N} &= F \cos \theta + (\frac{5}{13})(325 \text{ N}) + (600 \text{ N}) \cos 45^\circ \\ \stackrel{+}{\uparrow}(F_R)_y &= \Sigma F_y; \\ 0 &= F \sin \theta + (\frac{12}{13})(325 \text{ N}) - (600 \text{ N}) \sin 45^\circ \\ \tan \theta &= 0.6190 \quad \theta = 31.76^\circ = 31.8^\circ \angle \nwarrow \quad \text{Ans.} \\ F &= 236 \text{ N} \quad \text{Ans.} \end{aligned}$$

**F2-11.**  $\stackrel{+}{\rightarrow}(F_R)_x = \Sigma F_x;$

$$\begin{aligned} (80 \text{ lb}) \cos 45^\circ &= F \cos \theta + 50 \text{ lb} - (\frac{3}{5})(90 \text{ lb}) \\ \stackrel{+}{\uparrow}(F_R)_y &= \Sigma F_y; \\ -(80 \text{ lb}) \sin 45^\circ &= F \sin \theta - (\frac{4}{5})(90 \text{ lb}) \\ \tan \theta &= 0.2547 \quad \theta = 14.29^\circ = 14.3^\circ \angle \nwarrow \quad \text{Ans.} \\ F &= 62.5 \text{ lb} \quad \text{Ans.} \end{aligned}$$

**F2-12.**  $(F_R)_x = 15(\frac{4}{5}) + 0 + 15(\frac{4}{5}) = 24 \text{ kN} \rightarrow$

$$(F_R)_y = 15(\frac{3}{5}) + 20 - 15(\frac{3}{5}) = 20 \text{ kN} \uparrow$$

$$F_R = 31.2 \text{ kN}$$

$$\theta = 39.8^\circ$$

*Ans.*

*Ans.*

**F2-13.**  $F_x = 75 \cos 30^\circ \sin 45^\circ = 45.93 \text{ lb}$

$$F_y = 75 \cos 30^\circ \cos 45^\circ = 45.93 \text{ lb}$$

$$F_z = -75 \sin 30^\circ = -37.5 \text{ lb}$$

$$\alpha = \cos^{-1} \left( \frac{45.93}{75} \right) = 52.2^\circ$$

$$\beta = \cos^{-1} \left( \frac{45.93}{75} \right) = 52.2^\circ$$

*Ans.*

*Ans.*

$$\gamma = \cos^{-1} \left( \frac{-37.5}{75} \right) = 120^\circ$$

*Ans.*

**F2-14.**  $\cos \beta = \sqrt{1 - \cos^2 120^\circ - \cos^2 60^\circ} = \pm 0.7071$   
Require  $\beta = 135^\circ$ .

$$\mathbf{F} = F\mathbf{u}_F = (500 \text{ N})(-0.5\mathbf{i} - 0.7071\mathbf{j} + 0.5\mathbf{k}) \\ = \{-250\mathbf{i} - 354\mathbf{j} + 250\mathbf{k}\} \text{ N} \quad \text{Ans.}$$

**F2-15.**  $\cos^2 \alpha + \cos^2 135^\circ + \cos^2 120^\circ = 1$   
 $\alpha = 60^\circ$   
 $\mathbf{F} = F\mathbf{u}_F = (500 \text{ N})(0.5\mathbf{i} - 0.7071\mathbf{j} - 0.5\mathbf{k}) \\ = \{250\mathbf{i} - 354\mathbf{j} - 250\mathbf{k}\} \text{ N} \quad \text{Ans.}$

**F2-16.**  $F_z = (50 \text{ lb}) \sin 45^\circ = 35.36 \text{ lb}$   
 $F' = (50 \text{ lb}) \cos 45^\circ = 35.36 \text{ lb}$   
 $F_x = \left(\frac{3}{5}\right)(35.36 \text{ lb}) = 21.21 \text{ lb}$   
 $F_y = \left(\frac{4}{5}\right)(35.36 \text{ lb}) = 28.28 \text{ lb}$   
 $\mathbf{F} = \{-21.2\mathbf{i} + 28.3\mathbf{j} + 35.4\mathbf{k}\} \text{ lb} \quad \text{Ans.}$

**F2-17.**  $F_z = (750 \text{ N}) \sin 45^\circ = 530.33 \text{ N}$   
 $F' = (750 \text{ N}) \cos 45^\circ = 530.33 \text{ N}$   
 $F_x = (530.33 \text{ N}) \cos 60^\circ = 265.2 \text{ N}$   
 $F_y = (530.33 \text{ N}) \sin 60^\circ = 459.3 \text{ N}$   
 $\mathbf{F}_2 = \{265\mathbf{i} - 459\mathbf{j} + 530\mathbf{k}\} \text{ N} \quad \text{Ans.}$

**F2-18.**  $\mathbf{F}_1 = \left(\frac{4}{5}\right)(500 \text{ lb})\mathbf{j} + \left(\frac{3}{5}\right)(500 \text{ lb})\mathbf{k} \\ = \{400\mathbf{j} + 300\mathbf{k}\} \text{ lb}$   

$$\mathbf{F}_2 = [(800 \text{ lb}) \cos 45^\circ] \cos 30^\circ \mathbf{i} \\ + [(800 \text{ lb}) \cos 45^\circ] \sin 30^\circ \mathbf{j} \\ + (800 \text{ lb}) \sin 45^\circ (-\mathbf{k}) \\ = \{489.90\mathbf{i} + 282.84\mathbf{j} - 565.69\mathbf{k}\} \text{ lb}$$
  
 $\mathbf{F}_R = \mathbf{F}_1 + \mathbf{F}_2 = \{490\mathbf{i} + 683\mathbf{j} - 266\mathbf{k}\} \text{ lb} \quad \text{Ans.}$

**F2-19.**  $\mathbf{r}_{AB} = \{-6\mathbf{i} + 6\mathbf{j} + 3\mathbf{k}\} \text{ m} \quad \text{Ans.}$   
 $r_{AB} = \sqrt{(-6 \text{ m})^2 + (6 \text{ m})^2 + (3 \text{ m})^2} = 9 \text{ m} \quad \text{Ans.}$   
 $\alpha = 132^\circ, \beta = 48.2^\circ, \gamma = 70.5^\circ \quad \text{Ans.}$

**F2-20.**  $\mathbf{r}_{AB} = \{-4\mathbf{i} + 2\mathbf{j} + 4\mathbf{k}\} \text{ ft} \quad \text{Ans.}$   
 $r_{AB} = \sqrt{(-4 \text{ ft})^2 + (2 \text{ ft})^2 + (4 \text{ ft})^2} = 6 \text{ ft} \quad \text{Ans.}$   
 $\alpha = \cos^{-1}\left(\frac{-4 \text{ ft}}{6 \text{ ft}}\right) = 131.8^\circ$   
 $\theta = 180^\circ - 131.8^\circ = 48.2^\circ \quad \text{Ans.}$

**F2-21.**  $\mathbf{r}_{AB} = \{2\mathbf{i} + 3\mathbf{j} - 6\mathbf{k}\} \text{ m}$   
 $\mathbf{F}_{AB} = F_{AB}\mathbf{u}_{AB}$   

$$= (630 \text{ N})\left(\frac{2}{7}\mathbf{i} + \frac{3}{7}\mathbf{j} - \frac{6}{7}\mathbf{k}\right)$$
  
 $= \{180\mathbf{i} + 270\mathbf{j} - 540\mathbf{k}\} \text{ N} \quad \text{Ans.}$

**F2-22.**  $\mathbf{F} = F\mathbf{u}_{AB} = 900 \text{ N} \left( -\frac{4}{9}\mathbf{i} + \frac{7}{9}\mathbf{j} - \frac{4}{9}\mathbf{k} \right) \\ = \{-400\mathbf{i} + 700\mathbf{j} - 400\mathbf{k}\} \text{ N} \quad \text{Ans.}$

**F2-23.**  $\mathbf{F}_B = F_B\mathbf{u}_B$   

$$= (840 \text{ N})\left(\frac{3}{7}\mathbf{i} - \frac{2}{7}\mathbf{j} - \frac{6}{7}\mathbf{k}\right)$$
  
 $= \{360\mathbf{i} - 240\mathbf{j} - 720\mathbf{k}\} \text{ N}$   
 $\mathbf{F}_C = F_C\mathbf{u}_C$   

$$= (420 \text{ N})\left(\frac{2}{7}\mathbf{i} + \frac{3}{7}\mathbf{j} - \frac{6}{7}\mathbf{k}\right)$$
  
 $= \{120\mathbf{i} + 180\mathbf{j} - 360\mathbf{k}\} \text{ N}$   
 $F_R = \sqrt{(480 \text{ N})^2 + (-60 \text{ N})^2 + (-1080 \text{ N})^2}$   
 $= 1.18 \text{ kN} \quad \text{Ans.}$

**F2-24.**  $\mathbf{F}_B = F_B\mathbf{u}_B$   

$$= (600 \text{ lb})\left(-\frac{1}{3}\mathbf{i} + \frac{2}{3}\mathbf{j} - \frac{2}{3}\mathbf{k}\right)$$
  
 $= \{-200\mathbf{i} + 400\mathbf{j} - 400\mathbf{k}\} \text{ lb}$   
 $\mathbf{F}_C = F_C\mathbf{u}_C$   

$$= (490 \text{ lb})\left(-\frac{6}{7}\mathbf{i} + \frac{3}{7}\mathbf{j} - \frac{2}{7}\mathbf{k}\right)$$
  
 $= \{-420\mathbf{i} + 210\mathbf{j} - 140\mathbf{k}\} \text{ lb}$   
 $\mathbf{F}_R = \mathbf{F}_B + \mathbf{F}_C = \{-620\mathbf{i} + 610\mathbf{j} - 540\mathbf{k}\} \text{ lb} \quad \text{Ans.}$

**F2-25.**  $\mathbf{u}_{AO} = -\frac{1}{3}\mathbf{i} + \frac{2}{3}\mathbf{j} - \frac{2}{3}\mathbf{k}$   
 $\mathbf{u}_F = -0.5345\mathbf{i} + 0.8018\mathbf{j} + 0.2673\mathbf{k}$   
 $\theta = \cos^{-1}(\mathbf{u}_{AO} \cdot \mathbf{u}_F) = 57.7^\circ \quad \text{Ans.}$

**F2-26.**  $\mathbf{u}_{AB} = -\frac{3}{5}\mathbf{j} + \frac{4}{5}\mathbf{k}$   
 $\mathbf{u}_F = \frac{4}{5}\mathbf{i} - \frac{3}{5}\mathbf{j}$   
 $\theta = \cos^{-1}(\mathbf{u}_{AB} \cdot \mathbf{u}_F) = 68.9^\circ \quad \text{Ans.}$

**F2-27.**  $\mathbf{u}_{OA} = \frac{12}{13}\mathbf{i} + \frac{5}{13}\mathbf{j}$   
 $\mathbf{u}_{OA} \cdot \mathbf{j} = u_{OA}(1) \cos \theta$   
 $\cos \theta = \frac{5}{13}; \theta = 67.4^\circ \quad \text{Ans.}$

**F2-28.**  $\mathbf{u}_{OA} = \frac{12}{13}\mathbf{i} + \frac{5}{13}\mathbf{j}$   
 $\mathbf{F} = F\mathbf{u}_F = [650\mathbf{j}] \text{ N}$   
 $F_{OA} = \mathbf{F} \cdot \mathbf{u}_{OA} = 250 \text{ N}$   
 $\mathbf{F}_{OA} = F_{OA} \mathbf{u}_{OA} = \{231\mathbf{i} + 96.2\mathbf{j}\} \text{ N} \quad \text{Ans.}$

**F2-29.**  $\mathbf{F} = (400 \text{ N}) \frac{\{4\mathbf{i} + 1\mathbf{j} - 6\mathbf{k}\} \text{ m}}{\sqrt{(4 \text{ m})^2 + (1 \text{ m})^2 + (-6 \text{ m})^2}}$   
 $= \{219.78\mathbf{i} + 54.94\mathbf{j} - 329.67\mathbf{k}\} \text{ N}$   
 $\mathbf{u}_{AO} = \frac{\{-4\mathbf{j} - 6\mathbf{k}\} \text{ m}}{\sqrt{(-4 \text{ m})^2 + (-6 \text{ m})^2}}$   
 $= -0.5547\mathbf{j} - 0.8321\mathbf{k}$   
 $(F_{AO})_{\text{proj}} = \mathbf{F} \cdot \mathbf{u}_{AO} = 244 \text{ N}$  *Ans.*

**F2-30.**  $\mathbf{F} = [(-600 \text{ lb}) \cos 60^\circ] \sin 30^\circ \mathbf{i}$   
 $+ [(-600 \text{ lb}) \cos 60^\circ] \cos 30^\circ \mathbf{j}$   
 $+ [(600 \text{ lb}) \sin 60^\circ] \mathbf{k}$   
 $= \{-150\mathbf{i} + 259.81\mathbf{j} + 519.62\mathbf{k}\} \text{ lb}$   
 $\mathbf{u}_A = -\frac{2}{3}\mathbf{i} + \frac{2}{3}\mathbf{j} + \frac{1}{3}\mathbf{k}$   
 $(F_A)_{\text{par}} = \mathbf{F} \cdot \mathbf{u}_A = 446.41 \text{ lb} = 446 \text{ lb}$  *Ans.*  
 $(F_A)_{\text{per}} = \sqrt{(600 \text{ lb})^2 - (446.41 \text{ lb})^2}$   
 $= 401 \text{ lb}$  *Ans.*

**F2-31.**  $\mathbf{F} = 56 \text{ N} (\frac{3}{7}\mathbf{i} - \frac{6}{7}\mathbf{j} + \frac{2}{7}\mathbf{k})$   
 $= \{24\mathbf{i} - 48\mathbf{j} + 16\mathbf{k}\} \text{ N}$   
 $(F_{AO})_{\parallel} = \mathbf{F} \cdot \mathbf{u}_{AO} = (24\mathbf{i} - 48\mathbf{j} + 16\mathbf{k}) \cdot (\frac{3}{7}\mathbf{i} - \frac{6}{7}\mathbf{j} - \frac{2}{7}\mathbf{k})$   
 $= 46.86 \text{ N} = 46.9 \text{ N}$  *Ans.*  
 $(F_{AO})_{\perp} = \sqrt{F^2 - (F_{AO})_{\parallel}^2} = \sqrt{(56)^2 - (46.86)^2}$   
 $= 30.7 \text{ N}$  *Ans.*

## Chapter 3

**F3-1.**  $\stackrel{+}{\rightarrow} \Sigma F_x = 0; \frac{4}{5}F_{AC} - F_{AB} \cos 30^\circ = 0$   
 $\stackrel{+}{\uparrow} \Sigma F_y = 0; \frac{3}{5}F_{AC} + F_{AB} \sin 30^\circ - 550 = 0$   
 $F_{AB} = 478 \text{ lb}$  *Ans.*  
 $F_{AC} = 518 \text{ lb}$  *Ans.*

**F3-2.**  $\stackrel{+}{\uparrow} \Sigma F_y = 0; -2(1500) \sin \theta + 700 = 0$   
 $\theta = 13.5^\circ$   
 $L_{ABC} = 2\left(\frac{5 \text{ ft}}{\cos 13.5^\circ}\right) = 10.3 \text{ ft}$  *Ans.*

**F3-3.**  $\stackrel{+}{\rightarrow} \Sigma F_x = 0; T \cos \theta - T \cos \phi = 0$   
 $\phi = \theta$   
 $\stackrel{+}{\uparrow} \Sigma F_y = 0; 2T \sin \theta - 49.05 \text{ N} = 0$   
 $\theta = \tan^{-1}\left(\frac{0.15 \text{ m}}{0.2 \text{ m}}\right) = 36.87^\circ$   
 $T = 40.9 \text{ N}$  *Ans.*

**F3-4.**  $\stackrel{+}{\nearrow} \Sigma F_x = 0; \frac{4}{5}(F_{sp}) - 5(9.81) \sin 45^\circ = 0$   
 $F_{sp} = 43.35 \text{ N}$   
 $F_{sp} = k(l - l_0); 43.35 = 200(0.5 - l_0)$   
 $l_0 = 0.283 \text{ m}$  *Ans.*

**F3-5.**  $\stackrel{+}{\uparrow} \Sigma F_y = 0; (392.4 \text{ N}) \sin 30^\circ - m_A(9.81) = 0$   
 $m_A = 20 \text{ kg}$  *Ans.*

**F3-6.**  $\stackrel{+}{\uparrow} \Sigma F_y = 0; T_{AB} \sin 15^\circ - 10(9.81) \text{ N} = 0$   
 $T_{AB} = 379.03 \text{ N} = 379 \text{ N}$  *Ans.*  
 $\stackrel{+}{\rightarrow} \Sigma F_x = 0; T_{BC} - 379.03 \text{ N} \cos 15^\circ = 0$   
 $T_{BC} = 366.11 \text{ N} = 366 \text{ N}$  *Ans.*  
 $\stackrel{+}{\rightarrow} \Sigma F_x = 0; T_{CD} \cos \theta - 366.11 \text{ N} = 0$   
 $\stackrel{+}{\uparrow} \Sigma F_y = 0; T_{CD} \sin \theta - 15(9.81) \text{ N} = 0$   
 $T_{CD} = 395 \text{ N}$  *Ans.*  
 $\theta = 21.9^\circ$  *Ans.*

**F3-7.**  $\Sigma F_x = 0; \left[\left(\frac{3}{5}\right)F_3\right]\left(\frac{3}{5}\right) + 600 \text{ N} - F_2 = 0 \quad (1)$   
 $\Sigma F_y = 0; \left(\frac{4}{5}\right)F_1 - \left[\left(\frac{3}{5}\right)F_3\right]\left(\frac{4}{5}\right) = 0 \quad (2)$   
 $\Sigma F_z = 0; \left(\frac{4}{5}\right)F_3 + \left(\frac{3}{5}\right)F_1 - 900 \text{ N} = 0 \quad (3)$   
 $F_3 = 776 \text{ N}$  *Ans.*  
 $F_1 = 466 \text{ N}$  *Ans.*  
 $F_2 = 879 \text{ N}$  *Ans.*

**F3-8.**  $\Sigma F_z = 0; F_{AD}\left(\frac{4}{5}\right) - 900 = 0$   
 $F_{AD} = 1125 \text{ N} = 1.125 \text{ kN}$  *Ans.*  
 $\Sigma F_y = 0; F_{AC}\left(\frac{4}{5}\right) - 1125\left(\frac{3}{5}\right) = 0$   
 $F_{AC} = 843.75 \text{ N} = 844 \text{ N}$  *Ans.*  
 $\Sigma F_x = 0; F_{AB} - 843.75\left(\frac{3}{5}\right) = 0$   
 $F_{AB} = 506.25 \text{ N} = 506 \text{ N}$  *Ans.*

**F3-9.**  $F_{AD} = F_{AD}\left(\frac{\mathbf{r}_{AD}}{r_{AD}}\right) = \frac{1}{3}F_{AD}\mathbf{i} - \frac{2}{3}F_{AD}\mathbf{j} + \frac{2}{3}F_{AD}\mathbf{k}$   
 $\Sigma F_z = 0; \frac{2}{3}F_{AD} - 600 = 0$   
 $F_{AD} = 900 \text{ N}$  *Ans.*  
 $\Sigma F_y = 0; F_{AB} \cos 30^\circ - \frac{2}{3}(900) = 0$   
 $F_{AB} = 692.82 \text{ N} = 693 \text{ N}$  *Ans.*  
 $\Sigma F_x = 0; \frac{1}{3}(900) + 692.82 \sin 30^\circ - F_{AC} = 0$   
 $F_{AC} = 646.41 \text{ N} = 646 \text{ N}$  *Ans.*

**F3-10.**  $F_{AC} = F_{AC} \{ -\cos 60^\circ \sin 30^\circ \mathbf{i} + \cos 60^\circ \cos 30^\circ \mathbf{j} + \sin 60^\circ \mathbf{k} \}$   
 $= -0.25F_{AC}\mathbf{i} + 0.4330F_{AC}\mathbf{j} + 0.8660F_{AC}\mathbf{k}$   
 $F_{AD} = F_{AD} \{ \cos 120^\circ \mathbf{i} + \cos 120^\circ \mathbf{j} + \cos 45^\circ \mathbf{k} \}$   
 $= -0.5F_{AD}\mathbf{i} - 0.5F_{AD}\mathbf{j} + 0.7071F_{AD}\mathbf{k}$   
 $\Sigma F_y = 0; 0.4330F_{AC} - 0.5F_{AD} = 0$   
 $\Sigma F_z = 0; 0.8660F_{AC} + 0.7071F_{AD} - 300 = 0$   
 $F_{AD} = 175.74 \text{ lb} = 176 \text{ lb}$  *Ans.*  
 $F_{AC} = 202.92 \text{ lb} = 203 \text{ lb}$  *Ans.*  
 $\Sigma F_x = 0; F_{AB} - 0.25(202.92) - 0.5(175.74) = 0$   
 $F_{AB} = 138.60 \text{ lb} = 139 \text{ lb}$  *Ans.*

**F3-11.**  $\mathbf{F}_B = F_B \left( \frac{\mathbf{r}_{AB}}{r_{AB}} \right)$

$$= F_B \left[ \frac{\{-6\mathbf{i} + 3\mathbf{j} + 2\mathbf{k}\} \text{ ft}}{\sqrt{(-6 \text{ ft})^2 + (3 \text{ ft})^2 + (2 \text{ ft})^2}} \right]$$

$$= -\frac{6}{7}F_B\mathbf{i} + \frac{3}{7}F_B\mathbf{j} + \frac{2}{7}F_B\mathbf{k}$$

$\mathbf{F}_C = F_C \left( \frac{\mathbf{r}_{AC}}{r_{AC}} \right)$

$$= F_C \left[ \frac{\{-6\mathbf{i} - 2\mathbf{j} + 3\mathbf{k}\} \text{ ft}}{\sqrt{(-6 \text{ ft})^2 + (-2 \text{ ft})^2 + (3 \text{ ft})^2}} \right]$$

$$= -\frac{6}{7}F_C\mathbf{i} - \frac{2}{7}F_C\mathbf{j} + \frac{3}{7}F_C\mathbf{k}$$

$\mathbf{F}_D = F_D\mathbf{i}$   
 $\mathbf{W} = \{-150\mathbf{k}\} \text{ lb}$

$\Sigma F_x = 0; -\frac{6}{7}F_B - \frac{6}{7}F_C + F_D = 0 \quad (1)$

$\Sigma F_y = 0; \frac{3}{7}F_B - \frac{2}{7}F_C = 0 \quad (2)$

$\Sigma F_z = 0; \frac{2}{7}F_B + \frac{3}{7}F_C - 150 = 0 \quad (3)$

$F_B = 162 \text{ lb} \quad \text{Ans.}$

$F_C = 1.5(162 \text{ lb}) = 242 \text{ lb} \quad \text{Ans.}$

$F_D = 346.15 \text{ lb} = 346 \text{ lb} \quad \text{Ans.}$

## Chapter 4

**F4-1.**  $\zeta + M_O = -\left(\frac{4}{5}\right)(100 \text{ N})(2 \text{ m}) - \left(\frac{3}{5}\right)(100 \text{ N})(5 \text{ m})$   
 $= -460 \text{ N} \cdot \text{m} = 460 \text{ N} \cdot \text{m} \quad \text{Ans.}$

**F4-2.**  $\zeta + M_O = [(300 \text{ N}) \sin 30^\circ][0.4 \text{ m} + (0.3 \text{ m}) \cos 45^\circ] - [(300 \text{ N}) \cos 30^\circ][(0.3 \text{ m}) \sin 45^\circ]$   
 $= 36.7 \text{ N} \cdot \text{m} \quad \text{Ans.}$

**F4-3.**  $\zeta + M_O = (600 \text{ lb})(4 \text{ ft} + (3 \text{ ft}) \cos 45^\circ - 1 \text{ ft})$   
 $= 3.07 \text{ kip} \cdot \text{ft} \quad \text{Ans.}$

**F4-4.**  $\zeta + M_O = 50 \sin 60^\circ (0.1 + 0.2 \cos 45^\circ + 0.1) - 50 \cos 60^\circ (0.2 \sin 45^\circ)$   
 $= 11.2 \text{ N} \cdot \text{m} \quad \text{Ans.}$

**F4-5.**  $\zeta + M_O = 600 \sin 50^\circ (5) + 600 \cos 50^\circ (0.5)$   
 $= 2.49 \text{ kip} \cdot \text{ft} \quad \text{Ans.}$

**F4-6.**  $\zeta + M_O = 500 \sin 45^\circ (3 + 3 \cos 45^\circ) - 500 \cos 45^\circ (3 \sin 45^\circ)$   
 $= 1.06 \text{ kN} \cdot \text{m} \quad \text{Ans.}$

**F4-7.**  $\zeta + (M_R)_O = \Sigma Fd;$   
 $(M_R)_O = -(600 \text{ N})(1 \text{ m}) + (500 \text{ N})[3 \text{ m} + (2.5 \text{ m}) \cos 45^\circ] - (300 \text{ N})[(2.5 \text{ m}) \sin 45^\circ]$   
 $= 1254 \text{ N} \cdot \text{m} = 1.25 \text{ kN} \cdot \text{m} \quad \text{Ans.}$

**F4-8.**  $\zeta + (M_R)_O = \Sigma Fd;$   
 $(M_R)_O = \left[ \left( \frac{3}{5} \right) 500 \text{ N} \right] (0.425 \text{ m}) - \left[ \left( \frac{4}{5} \right) 500 \text{ N} \right] (0.25 \text{ m}) - [(600 \text{ N}) \cos 60^\circ] (0.25 \text{ m}) - [(600 \text{ N}) \sin 60^\circ] (0.425 \text{ m})$   
 $= -268 \text{ N} \cdot \text{m} = 268 \text{ N} \cdot \text{m} \quad \text{Ans.}$

**F4-9.**  $\zeta + (M_R)_O = \Sigma Fd;$   
 $(M_R)_O = (300 \cos 30^\circ \text{ lb})(6 \text{ ft} + 6 \sin 30^\circ \text{ ft}) - (300 \sin 30^\circ \text{ lb})(6 \cos 30^\circ \text{ ft}) + (200 \text{ lb})(6 \cos 30^\circ \text{ ft})$   
 $= 2.60 \text{ kip} \cdot \text{ft} \quad \text{Ans.}$

**F4-10.**  $\mathbf{F} = F\mathbf{u}_{AB} = 500 \text{ N} \left( \frac{4}{5}\mathbf{i} - \frac{3}{5}\mathbf{j} \right) = \{400\mathbf{i} - 300\mathbf{j}\} \text{ N}$   
 $\mathbf{M}_O = \mathbf{r}_{OA} \times \mathbf{F} = \{3\mathbf{j}\} \text{ m} \times \{400\mathbf{i} - 300\mathbf{j}\} \text{ N}$   
 $= \{-1200\mathbf{k}\} \text{ N} \cdot \text{m} \quad \text{Ans.}$

or

$\mathbf{M}_O = \mathbf{r}_{OB} \times \mathbf{F} = \{4\mathbf{i}\} \text{ m} \times \{400\mathbf{i} - 300\mathbf{j}\} \text{ N}$   
 $= \{-1200\mathbf{k}\} \text{ N} \cdot \text{m} \quad \text{Ans.}$

**F4-11.**  $\mathbf{F} = F\mathbf{u}_{BC}$   
 $= 120 \text{ lb} \left[ \frac{\{4\mathbf{i} - 4\mathbf{j} - 2\mathbf{k}\} \text{ ft}}{\sqrt{(4 \text{ ft})^2 + (-4 \text{ ft})^2 + (-2 \text{ ft})^2}} \right]$   
 $= \{80\mathbf{i} - 80\mathbf{j} - 40\mathbf{k}\} \text{ lb}$   
 $\mathbf{M}_O = \mathbf{r}_C \times \mathbf{F} = \begin{vmatrix} \mathbf{i} & \mathbf{j} & \mathbf{k} \\ 5 & 0 & 0 \\ 80 & -80 & -40 \end{vmatrix}$   
 $= \{200\mathbf{j} - 400\mathbf{k}\} \text{ lb} \cdot \text{ft} \quad \text{Ans.}$

or

$\mathbf{M}_O = \mathbf{r}_B \times \mathbf{F} = \begin{vmatrix} \mathbf{i} & \mathbf{j} & \mathbf{k} \\ 1 & 4 & 2 \\ 80 & -80 & -40 \end{vmatrix}$   
 $= \{200\mathbf{j} - 400\mathbf{k}\} \text{ lb} \cdot \text{ft} \quad \text{Ans.}$

**F4-12.**  $\mathbf{F}_R = \mathbf{F}_1 + \mathbf{F}_2$   
 $= \{(100 - 200)\mathbf{i} + (-120 + 250)\mathbf{j} + (75 + 100)\mathbf{k}\} \text{ lb}$   
 $= \{-100\mathbf{i} + 130\mathbf{j} + 175\mathbf{k}\} \text{ lb}$   
 $(\mathbf{M}_R)_O = \mathbf{r}_A \times \mathbf{F}_R = \begin{vmatrix} \mathbf{i} & \mathbf{j} & \mathbf{k} \\ 4 & 5 & 3 \\ -100 & 130 & 175 \end{vmatrix}$   
 $= \{485\mathbf{i} - 1000\mathbf{j} + 1020\mathbf{k}\} \text{ lb} \cdot \text{ft} \quad \text{Ans.}$

**F4-13.**  $M_x = \mathbf{i} \cdot (\mathbf{r}_{OB} \times \mathbf{F}) = \begin{vmatrix} 1 & 0 & 0 \\ 0.3 & 0.4 & -0.2 \\ 300 & -200 & 150 \end{vmatrix}$   
 $= 20 \text{ N} \cdot \text{m}$  *Ans.*

**F4-14.**  $\mathbf{u}_{OA} = \frac{\mathbf{r}_A}{r_A} = \frac{\{0.3\mathbf{i} + 0.4\mathbf{j}\} \text{ m}}{\sqrt{(0.3 \text{ m})^2 + (0.4 \text{ m})^2}} = 0.6\mathbf{i} + 0.8\mathbf{j}$   
 $M_{OA} = \mathbf{u}_{OA} \cdot (\mathbf{r}_{AB} \times \mathbf{F}) = \begin{vmatrix} 0.6 & 0.8 & 0 \\ 0 & 0 & -0.2 \\ 300 & -200 & 150 \end{vmatrix}$   
 $= -72 \text{ N} \cdot \text{m}$  *Ans.*

$|M_{OA}| = 72 \text{ N} \cdot \text{m}$

**F4-15.** Scalar Analysis  
The magnitudes of the force components are  
 $F_x = |200 \cos 120^\circ| = 100 \text{ N}$   
 $F_y = 200 \cos 60^\circ = 100 \text{ N}$   
 $F_z = 200 \cos 45^\circ = 141.42 \text{ N}$   
 $M_x = -F_y(z) + F_z(y)$   
 $= -(100 \text{ N})(0.25 \text{ m}) + (141.42 \text{ N})(0.3 \text{ m})$   
 $= 17.4 \text{ N} \cdot \text{m}$  *Ans.*

Vector Analysis

$M_x = \begin{vmatrix} 1 & 0 & 0 \\ 0 & 0.3 & 0.25 \\ -100 & 100 & 141.42 \end{vmatrix} = 17.4 \text{ N} \cdot \text{m}$  *Ans.*

**F4-16.**  $M_y = \mathbf{j} \cdot (\mathbf{r}_A \times \mathbf{F}) = \begin{vmatrix} 0 & 1 & 0 \\ -3 & -4 & 2 \\ 30 & -20 & 50 \end{vmatrix}$   
 $= 210 \text{ N} \cdot \text{m}$  *Ans.*

**F4-17.**  $\mathbf{u}_{AB} = \frac{\mathbf{r}_{AB}}{r_{AB}} = \frac{\{-4\mathbf{i} + 3\mathbf{j}\} \text{ ft}}{\sqrt{(-4 \text{ ft})^2 + (3 \text{ ft})^2}} = -0.8\mathbf{i} + 0.6\mathbf{j}$   
 $M_{AB} = \mathbf{u}_{AB} \cdot (\mathbf{r}_{AC} \times \mathbf{F})$   
 $= \begin{vmatrix} -0.8 & 0.6 & 0 \\ 0 & 0 & 2 \\ 50 & -40 & 20 \end{vmatrix} = -4 \text{ lb} \cdot \text{ft}$

$\mathbf{M}_{AB} = M_{AB}\mathbf{u}_{AB} = \{3.2\mathbf{i} - 2.4\mathbf{j}\} \text{ lb} \cdot \text{ft}$  *Ans.*

**F4-18.** Scalar Analysis  
The magnitudes of the force components are  
 $F_x = \left(\frac{3}{5}\right)\left[\frac{4}{5}(500)\right] = 240 \text{ N}$   
 $F_y = \frac{4}{5}\left[\frac{4}{5}(500)\right] = 320 \text{ N}$

$F_z = \frac{3}{5}(500) = 300 \text{ N}$   
 $M_x = -320(3) + 300(2) = -360 \text{ N} \cdot \text{m}$  *Ans.*  
 $M_y = -240(3) - 300(-2) = -120 \text{ N} \cdot \text{m}$  *Ans.*  
 $M_z = 240(2) - 320(2) = -160 \text{ N} \cdot \text{m}$  *Ans.*

Vector Analysis

$\mathbf{F} = \{-240\mathbf{i} + 320\mathbf{j} + 300\mathbf{k}\} \text{ N}$

$\mathbf{r}_{OA} = \{-2\mathbf{i} + 2\mathbf{j} + 3\mathbf{k}\} \text{ m}$

$M_x = \mathbf{i} \cdot (\mathbf{r}_{OA} \times \mathbf{F}) = -360 \text{ N} \cdot \text{m}$

$M_y = \mathbf{j} \cdot (\mathbf{r}_{OA} \times \mathbf{F}) = -120 \text{ N} \cdot \text{m}$

$M_z = \mathbf{k} \cdot (\mathbf{r}_{OA} \times \mathbf{F}) = -160 \text{ N} \cdot \text{m}$

**F4-19.**  $\zeta + M_{C_R} = \Sigma M_A = 400(3) - 400(5) + 300(5)$   
 $+ 200(0.2) = 740 \text{ N} \cdot \text{m}$  *Ans.*

Also,

$\zeta + M_{C_R} = 300(5) - 400(2) + 200(0.2)$   
 $= 740 \text{ N} \cdot \text{m}$  *Ans.*

**F4-20.**  $\zeta + M_{C_R} = 300(4) + 200(4) + 150(4)$   
 $= 2600 \text{ lb} \cdot \text{ft}$  *Ans.*

**F4-21.**  $\zeta + (M_B)_R = \Sigma M_B$   
 $-1.5 \text{ kN} \cdot \text{m} = (2 \text{ kN})(0.3 \text{ m}) - F(0.9 \text{ m})$

$F = 2.33 \text{ kN}$  *Ans.*

**F4-22.**  $\zeta + M_C = 10\left(\frac{3}{5}\right)(2) - 10\left(\frac{4}{5}\right)(4) = -20 \text{ kN} \cdot \text{m}$   
 $= 20 \text{ kN} \cdot \text{m}$  *Ans.*

**F4-23.**  $\mathbf{u}_1 = \frac{\mathbf{r}_1}{r_1} = \frac{\{-2\mathbf{i} + 2\mathbf{j} + 3.5\mathbf{k}\} \text{ ft}}{\sqrt{(-2 \text{ ft})^2 + (2 \text{ ft})^2 + (3.5 \text{ ft})^2}}$   
 $= -\frac{2}{4.5}\mathbf{i} + \frac{2}{4.5}\mathbf{j} + \frac{3.5}{4.5}\mathbf{k}$

$\mathbf{u}_2 = -\mathbf{k}$

$\mathbf{u}_3 = \frac{1.5}{2.5}\mathbf{i} - \frac{2}{2.5}\mathbf{j}$

$(\mathbf{M}_c)_1 = (M_c)_1 \mathbf{u}_1$   
 $= (450 \text{ lb} \cdot \text{ft})\left(-\frac{2}{4.5}\mathbf{i} + \frac{2}{4.5}\mathbf{j} + \frac{3.5}{4.5}\mathbf{k}\right)$   
 $= \{-200\mathbf{i} + 200\mathbf{j} + 350\mathbf{k}\} \text{ lb} \cdot \text{ft}$

$(\mathbf{M}_c)_2 = (M_c)_2 \mathbf{u}_2 = (250 \text{ lb} \cdot \text{ft})(-\mathbf{k})$   
 $= \{-250\mathbf{k}\} \text{ lb} \cdot \text{ft}$

$(\mathbf{M}_c)_3 = (M_c)_3 \mathbf{u}_3 = (300 \text{ lb} \cdot \text{ft})\left(\frac{1.5}{2.5}\mathbf{i} - \frac{2}{2.5}\mathbf{j}\right)$   
 $= \{180\mathbf{i} - 240\mathbf{j}\} \text{ lb} \cdot \text{ft}$

$(\mathbf{M}_c)_R = \Sigma M_c;$

$(\mathbf{M}_c)_R = \{-20\mathbf{i} - 40\mathbf{j} + 100\mathbf{k}\} \text{ lb} \cdot \text{ft}$  *Ans.*

**F4-24.**  $\mathbf{F}_B = \left(\frac{4}{5}\right)(450 \text{ N})\mathbf{j} - \left(\frac{3}{5}\right)(450 \text{ N})\mathbf{k}$   
 $= \{360\mathbf{j} - 270\mathbf{k}\} \text{ N}$

$$\mathbf{M}_c = \mathbf{r}_{AB} \times \mathbf{F}_B = \begin{vmatrix} \mathbf{i} & \mathbf{j} & \mathbf{k} \\ 0.4 & 0 & 0 \\ 0 & 360 & -270 \end{vmatrix}$$

$= \{108\mathbf{j} + 144\mathbf{k}\} \text{ N} \cdot \text{m}$

Also,

$\mathbf{M}_c = (\mathbf{r}_A \times \mathbf{F}_A) + (\mathbf{r}_B \times \mathbf{F}_B)$

$$= \begin{vmatrix} \mathbf{i} & \mathbf{j} & \mathbf{k} \\ 0 & 0 & 0.3 \\ 0 & -360 & 270 \end{vmatrix} + \begin{vmatrix} \mathbf{i} & \mathbf{j} & \mathbf{k} \\ 0.4 & 0 & 0.3 \\ 0 & 360 & -270 \end{vmatrix}$$

$= \{108\mathbf{j} + 144\mathbf{k}\} \text{ N} \cdot \text{m}$

*Ans.*

**F4-25.**  $\stackrel{+}{\rightarrow} F_{Rx} = \Sigma F_x; F_{Rx} = 200 - \frac{3}{5}(100) = 140 \text{ lb}$

$+ \downarrow F_{Ry} = \Sigma F_y; F_{Ry} = 150 - \frac{4}{5}(100) = 70 \text{ lb}$

$F_R = \sqrt{140^2 + 70^2} = 157 \text{ lb}$

$\theta = \tan^{-1}\left(\frac{70}{140}\right) = 26.6^\circ \curvearrowright$

*Ans.*

*Ans.*

$\zeta + M_{A_R} = \Sigma M_A;$

$M_{A_R} = \frac{3}{5}(100)(4) - \frac{4}{5}(100)(6) + 150(3)$

$M_{R_A} = 210 \text{ lb} \cdot \text{ft}$

*Ans.*

**F4-26.**  $\stackrel{+}{\rightarrow} F_{Rx} = \Sigma F_x; F_{Rx} = \frac{4}{5}(50) = 40 \text{ N}$

$+ \downarrow F_{Ry} = \Sigma F_y; F_{Ry} = 40 + 30 + \frac{3}{5}(50)$

$= 100 \text{ N}$

$F_R = \sqrt{(40)^2 + (100)^2} = 108 \text{ N}$

$\theta = \tan^{-1}\left(\frac{100}{40}\right) = 68.2^\circ \curvearrowright$

*Ans.*

*Ans.*

$\zeta + M_{A_R} = \Sigma M_A;$

$M_{A_R} = 30(3) + \frac{3}{5}(50)(6) + 200$

$= 470 \text{ N} \cdot \text{m}$

*Ans.*

**F4-27.**  $\stackrel{+}{\rightarrow} (F_R)_x = \Sigma F_x;$

$(F_R)_x = 900 \sin 30^\circ = 450 \text{ N} \rightarrow$

$+ \uparrow (F_R)_y = \Sigma F_y;$

$(F_R)_y = -900 \cos 30^\circ - 300$

$= -1079.42 \text{ N} = 1079.42 \text{ N} \downarrow$

$F_R = \sqrt{450^2 + 1079.42^2}$

$= 1169.47 \text{ N} = 1.17 \text{ kN}$

*Ans.*

$\theta = \tan^{-1}\left(\frac{1079.42}{450}\right) = 67.4^\circ \curvearrowright$

*Ans.*

$\zeta + (M_R)_A = \Sigma M_A;$

$(M_R)_A = 300 - 900 \cos 30^\circ (0.75) - 300(2.25)$

$= -959.57 \text{ N} \cdot \text{m}$

$= 960 \text{ N} \cdot \text{m} \curvearrowright$

*Ans.*

**F4-28.**  $\stackrel{+}{\rightarrow} (F_R)_x = \Sigma F_x;$

$(F_R)_x = 150\left(\frac{3}{5}\right) + 50 - 100\left(\frac{4}{5}\right) = 60 \text{ lb} \rightarrow$

$+ \uparrow (F_R)_y = \Sigma F_y;$

$(F_R)_y = -150\left(\frac{4}{5}\right) - 100\left(\frac{3}{5}\right)$

$= -180 \text{ lb} = 180 \text{ lb} \downarrow$

$F_R = \sqrt{60^2 + 180^2} = 189.74 \text{ lb} = 190 \text{ lb} \quad \text{Ans.}$

$\theta = \tan^{-1}\left(\frac{180}{60}\right) = 71.6^\circ \curvearrowright$

*Ans.*

$\zeta + (M_R)_A = \Sigma M_A;$

$(M_R)_A = 100\left(\frac{4}{5}\right)(1) - 100\left(\frac{3}{5}\right)(6) - 150\left(\frac{4}{5}\right)(3)$

$= -640 = 640 \text{ lb} \cdot \text{ft} \curvearrowright$

*Ans.*

**F4-29.**  $\mathbf{F}_R = \Sigma \mathbf{F};$

$F_R = \mathbf{F}_1 + \mathbf{F}_2$

$= (-300\mathbf{i} + 150\mathbf{j} + 200\mathbf{k}) + (-450\mathbf{k})$

$= \{-300\mathbf{i} + 150\mathbf{j} - 250\mathbf{k}\} \text{ N}$

*Ans.*

$\mathbf{r}_{OA} = (2 - 0)\mathbf{j} = \{2\mathbf{j}\} \text{ m}$

$\mathbf{r}_{OB} = (-1.5 - 0)\mathbf{i} + (2 - 0)\mathbf{j} + (1 - 0)\mathbf{k}$

$= \{-1.5\mathbf{i} + 2\mathbf{j} + 1\mathbf{k}\} \text{ m}$

$(\mathbf{M}_R)_O = \Sigma \mathbf{M};$

$(\mathbf{M}_R)_O = \mathbf{r}_{OB} \times \mathbf{F}_1 + \mathbf{r}_{OA} \times \mathbf{F}_2$

$$= \begin{vmatrix} \mathbf{i} & \mathbf{j} & \mathbf{k} \\ -1.5 & 2 & 1 \\ -300 & 150 & 200 \end{vmatrix} + \begin{vmatrix} \mathbf{i} & \mathbf{j} & \mathbf{k} \\ 0 & 2 & 0 \\ 0 & 0 & -450 \end{vmatrix}$$

$= \{-650\mathbf{i} + 375\mathbf{k}\} \text{ N} \cdot \text{m}$

*Ans.*

**F4-30.**  $\mathbf{F}_1 = \{-100\mathbf{j}\} \text{ N}$

$\mathbf{F}_2 = (200 \text{ N}) \left[ \frac{\{-0.4\mathbf{i} - 0.3\mathbf{k}\} \text{ m}}{\sqrt{(-0.4 \text{ m})^2 + (-0.3 \text{ m})^2}} \right]$

$= \{-160\mathbf{i} - 120\mathbf{k}\} \text{ N}$

$\mathbf{M}_c = \{-75\mathbf{i}\} \text{ N} \cdot \text{m}$

$\mathbf{F}_R = \{-160\mathbf{i} - 100\mathbf{j} - 120\mathbf{k}\} \text{ N}$

*Ans.*

$(\mathbf{M}_R)_O = (0.3\mathbf{k}) \times (-100\mathbf{j})$

$$+ \begin{vmatrix} \mathbf{i} & \mathbf{j} & \mathbf{k} \\ 0 & 0.5 & 0.3 \\ -160 & 0 & -120 \end{vmatrix} + (-75\mathbf{i})$$

$= \{-105\mathbf{i} - 48\mathbf{j} + 80\mathbf{k}\} \text{ N} \cdot \text{m}$

*Ans.*

**F4-31.**  $+ \downarrow F_R = \Sigma F_y; F_R = 500 + 250 + 500$

$= 1250 \text{ lb}$

*Ans.*

$\zeta + F_R x = \Sigma M_O;$

$1250(x) = 500(3) + 250(6) + 500(9)$

$x = 6 \text{ ft}$

*Ans.*

**F4-32.**  $\rightarrow (F_R)_x = \Sigma F_x;$   
 $(F_R)_x = 100\left(\frac{3}{5}\right) + 50 \sin 30^\circ = 85 \text{ lb} \rightarrow$   
 $+ \uparrow (F_R)_y = \Sigma F_y;$   
 $(F_R)_y = 200 + 50 \cos 30^\circ - 100\left(\frac{4}{5}\right)$   
 $= 163.30 \text{ lb} \uparrow$   
 $F_R = \sqrt{85^2 + 163.30^2} = 184 \text{ lb}$   
 $\theta = \tan^{-1}\left(\frac{163.30}{85}\right) = 62.5^\circ \angle$  *Ans.*

$\zeta + (M_R)_A = \Sigma M_A;$   
 $163.30(d) = 200(3) - 100\left(\frac{4}{5}\right)(6) + 50 \cos 30^\circ(9)$   
 $d = 3.12 \text{ ft}$  *Ans.*

**F4-33.**  $\rightarrow (F_R)_x = \Sigma F_x;$   
 $(F_R)_x = 15\left(\frac{4}{5}\right) = 12 \text{ kN} \rightarrow$   
 $+ \uparrow (F_R)_y = \Sigma F_y;$   
 $(F_R)_y = -20 + 15\left(\frac{3}{5}\right) = -11 \text{ kN} = 11 \text{ kN} \downarrow$   
 $F_R = \sqrt{12^2 + 11^2} = 16.3 \text{ kN}$  *Ans.*  
 $\theta = \tan^{-1}\left(\frac{11}{12}\right) = 42.5^\circ \nwarrow$  *Ans.*

$\zeta + (M_R)_A = \Sigma M_A;$   
 $-11(d) = -20(2) - 15\left(\frac{4}{5}\right)(2) + 15\left(\frac{3}{5}\right)(6)$   
 $d = 0.909 \text{ m}$  *Ans.*

**F4-34.**  $\rightarrow (F_R)_x = \Sigma F_x;$   
 $(F_R)_x = \left(\frac{3}{5}\right) 5 \text{ kN} - 8 \text{ kN}$   
 $= -5 \text{ kN} = 5 \text{ kN} \leftarrow$   
 $+ \uparrow (F_R)_y = \Sigma F_y;$   
 $(F_R)_y = -6 \text{ kN} - \left(\frac{4}{5}\right) 5 \text{ kN}$   
 $= -10 \text{ kN} = 10 \text{ kN} \downarrow$   
 $F_R = \sqrt{5^2 + 10^2} = 11.2 \text{ kN}$  *Ans.*  
 $\theta = \tan^{-1}\left(\frac{10 \text{ kN}}{5 \text{ kN}}\right) = 63.4^\circ \angle$  *Ans.*

$\zeta + (M_R)_A = \Sigma M_A;$   
 $5 \text{ kN}(d) = 8 \text{ kN}(3 \text{ m}) - 6 \text{ kN}(0.5 \text{ m})$   
 $- \left[\left(\frac{4}{5}\right) 5 \text{ kN}\right](2 \text{ m})$   
 $- \left[\left(\frac{3}{5}\right) 5 \text{ kN}\right](4 \text{ m})$   
 $d = 0.2 \text{ m}$  *Ans.*

**F4-35.**  $\downarrow F_R = \Sigma F_z; F_R = 400 + 500 - 100$   
 $= 800 \text{ N}$   
 $M_{Rx} = \Sigma M_x; -800y = -400(4) - 500(4)$   
 $y = 4.50 \text{ m}$   
 $M_{Ry} = \Sigma M_y; 800x = 500(4) - 100(3)$   
 $x = 2.125 \text{ m}$  *Ans.*

**F4-36.**  $\downarrow F_R = \Sigma F_z;$   
 $F_R = 200 + 200 + 100 + 100$   
 $= 600 \text{ N}$  *Ans.*

$\zeta + M_{Rx} = \Sigma M_x;$   
 $-600y = 200(1) + 200(1) + 100(3) - 100(3)$   
 $y = -0.667 \text{ m}$  *Ans.*

$\zeta + M_{Ry} = \Sigma M_y;$   
 $600x = 100(3) + 100(3) + 200(2) - 200(3)$   
 $x = 0.667 \text{ m}$  *Ans.*

**F4-37.**  $\uparrow F_R = \Sigma F_y;$   
 $-F_R = -6(1.5) - 9(3) - 3(1.5)$   
 $F_R = 40.5 \text{ kN} \downarrow$  *Ans.*

$\zeta + (M_R)_A = \Sigma M_A;$   
 $-40.5(d) = 6(1.5)(0.75)$   
 $-9(3)(1.5) - 3(1.5)(3.75)$   
 $d = 1.25 \text{ m}$  *Ans.*

**F4-38.**  $F_R = \frac{1}{2}(6)(150) + 8(150) = 1650 \text{ lb}$  *Ans.*

$\zeta + M_{A_R} = \Sigma M_A;$   
 $1650d = \left[\frac{1}{2}(6)(150)\right](4) + [8(150)](10)$   
 $d = 8.36 \text{ ft}$  *Ans.*

**F4-39.**  $\uparrow F_R = \Sigma F_y;$   
 $-F_R = -\frac{1}{2}(6)(3) - \frac{1}{2}(6)(6)$   
 $F_R = 27 \text{ kN} \downarrow$  *Ans.*

$\zeta + (M_R)_A = \Sigma M_A;$   
 $-27(d) = \frac{1}{2}(6)(3)(1) - \frac{1}{2}(6)(6)(2)$   
 $d = 1 \text{ m}$  *Ans.*

**F4-40.**  $\downarrow F_R = \Sigma F_y;$   
 $F_R = \frac{1}{2}(50)(6) + 150(6) + 500$   
 $= 1550 \text{ lb}$  *Ans.*

$\zeta + M_{A_R} = \Sigma M_A;$   
 $1550d = \left[\frac{1}{2}(50)(6)\right](4) + [150(6)](3) + 500(9)$   
 $d = 5.03 \text{ ft}$  *Ans.*

**F4-41.**  $\uparrow F_R = \Sigma F_y;$   
 $-F_R = -\frac{1}{2}(3)(4.5) - 3(6)$   
 $F_R = 24.75 \text{ kN} \downarrow$  *Ans.*

$\zeta + (M_R)_A = \Sigma M_A;$   
 $-24.75(d) = -\frac{1}{2}(3)(4.5)(1.5) - 3(6)(3)$   
 $d = 2.59 \text{ m}$  *Ans.*

**F4-42.**  $F_R = \int w(x) dx = \int_0^4 2.5x^3 dx = 160 \text{ N}$

$$\zeta + \sum M_{A_R} = \sum M_A;$$

$$x = \frac{\int xw(x) dx}{\int w(x) dx} = \frac{\int_0^4 2.5x^4 dx}{160} = 3.20 \text{ m } \text{Ans.}$$

## Chapter 5

**F5-1.**  $\stackrel{+}{\Sigma} F_x = 0; -A_x + 500\left(\frac{3}{5}\right) = 0$   
 $A_x = 300 \text{ lb } \text{Ans.}$

 $\zeta + \sum M_A = 0; B_y(10) - 500\left(\frac{4}{5}\right)(5) - 600 = 0$   
 $B_y = 260 \text{ lb } \text{Ans.}$ 
 $+ \uparrow \Sigma F_y = 0; A_y + 260 - 500\left(\frac{4}{5}\right) = 0$   
 $A_y = 140 \text{ lb } \text{Ans.}$

**F5-2.**  $\zeta + \sum M_A = 0;$   
 $F_{CD} \sin 45^\circ(1.5 \text{ m}) - 4 \text{ kN}(3 \text{ m}) = 0$   
 $F_{CD} = 11.31 \text{ kN} = 11.3 \text{ kN } \text{Ans.}$

 $\stackrel{+}{\Sigma} F_x = 0; A_x + (11.31 \text{ kN}) \cos 45^\circ = 0$   
 $A_x = -8 \text{ kN} = 8 \text{ kN} \leftarrow \text{Ans.}$ 
 $+ \uparrow \Sigma F_y = 0;$   
 $A_y + (11.31 \text{ kN}) \sin 45^\circ - 4 \text{ kN} = 0$   
 $A_y = -4 \text{ kN} = 4 \text{ kN} \downarrow \text{Ans.}$

**F5-3.**  $\zeta + \sum M_A = 0;$   
 $N_B[6 \text{ m} + (6 \text{ m}) \cos 45^\circ]$   
 $- 10 \text{ kN}[2 \text{ m} + (6 \text{ m}) \cos 45^\circ]$   
 $- 5 \text{ kN}(4 \text{ m}) = 0$   
 $N_B = 8.047 \text{ kN} = 8.05 \text{ kN } \text{Ans.}$

 $\stackrel{+}{\Sigma} F_x = 0;$   
 $(5 \text{ kN}) \cos 45^\circ - A_x = 0$   
 $A_x = 3.54 \text{ kN } \text{Ans.}$ 
 $+ \uparrow \Sigma F_y = 0;$   
 $A_y + 8.047 \text{ kN} - (5 \text{ kN}) \sin 45^\circ - 10 \text{ kN} = 0$   
 $A_y = 5.49 \text{ kN } \text{Ans.}$

**F5-4.**  $\stackrel{+}{\Sigma} F_x = 0; -A_x + 400 \cos 30^\circ = 0$   
 $A_x = 346 \text{ N } \text{Ans.}$ 
 $+ \uparrow \Sigma F_y = 0;$   
 $A_y - 200 - 200 - 200 - 400 \sin 30^\circ = 0$   
 $A_y = 800 \text{ N } \text{Ans.}$

 $\zeta + \sum M_A = 0;$   
 $M_A - 200(2.5) - 200(3.5) - 200(4.5)$   
 $- 400 \sin 30^\circ(4.5) - 400 \cos 30^\circ(3 \sin 60^\circ) = 0$   
 $M_A = 3.90 \text{ kN} \cdot \text{m } \text{Ans.}$

**F5-5.**  $\zeta + \sum M_A = 0;$   
 $N_C(0.7 \text{ m}) - [25(9.81) \text{ N}] (0.5 \text{ m}) \cos 30^\circ = 0$   
 $N_C = 151.71 \text{ N} = 152 \text{ N } \text{Ans.}$

 $\stackrel{+}{\Sigma} F_x = 0;$   
 $T_{AB} \cos 15^\circ - (151.71 \text{ N}) \cos 60^\circ = 0$   
 $T_{AB} = 78.53 \text{ N} = 78.5 \text{ N } \text{Ans.}$ 
 $+ \uparrow \Sigma F_y = 0;$   
 $F_A + (78.53 \text{ N}) \sin 15^\circ$   
 $+ (151.71 \text{ N}) \sin 60^\circ - 25(9.81) \text{ N} = 0$   
 $F_A = 93.5 \text{ N } \text{Ans.}$

**F5-6.**  $\stackrel{+}{\Sigma} F_x = 0;$   
 $N_C \sin 30^\circ - (250 \text{ N}) \sin 60^\circ = 0$   
 $N_C = 433.0 \text{ N} = 433 \text{ N } \text{Ans.}$

 $\zeta + \sum M_B = 0;$   
 $-N_A \sin 30^\circ(0.15 \text{ m}) - 433.0 \text{ N}(0.2 \text{ m})$   
 $+ [(250 \text{ N}) \cos 30^\circ](0.6 \text{ m}) = 0$   
 $N_A = 577.4 \text{ N} = 577 \text{ N } \text{Ans.}$ 
 $+ \uparrow \Sigma F_y = 0;$   
 $N_B - 577.4 \text{ N} + (433.0 \text{ N}) \cos 30^\circ$   
 $- (250 \text{ N}) \cos 60^\circ = 0$   
 $N_B = 327 \text{ N } \text{Ans.}$

**F5-7.**  $\Sigma F_z = 0;$   
 $T_A + T_B + T_C - 200 - 500 = 0$   
 $\Sigma M_x = 0;$   
 $T_A(3) + T_C(3) - 500(1.5) - 200(3) = 0$   
 $\Sigma M_y = 0;$   
 $-T_B(4) - T_C(4) + 500(2) + 200(2) = 0$   
 $T_A = 350 \text{ lb}, T_B = 250 \text{ lb}, T_C = 100 \text{ lb } \text{Ans.}$

**F5-8.**  $\Sigma M_y = 0;$   
 $600 \text{ N}(0.2 \text{ m}) + 900 \text{ N}(0.6 \text{ m}) - F_A(1 \text{ m}) = 0$   
 $F_A = 660 \text{ N } \text{Ans.}$

 $\Sigma M_x = 0;$   
 $D_z(0.8 \text{ m}) - 600 \text{ N}(0.5 \text{ m}) - 900 \text{ N}(0.1 \text{ m}) = 0$   
 $D_z = 487.5 \text{ N } \text{Ans.}$ 
 $\Sigma F_x = 0; D_x = 0 \text{ Ans.}$ 
 $\Sigma F_y = 0; D_y = 0 \text{ Ans.}$ 
 $\Sigma F_z = 0;$   
 $T_{BC} + 660 \text{ N} + 487.5 \text{ N} - 900 \text{ N} - 600 \text{ N} = 0$   
 $T_{BC} = 352.5 \text{ N } \text{Ans.}$

- F5-9.**  $\Sigma F_y = 0; 400 \text{ N} + C_y = 0;$   
 $C_y = -400 \text{ N}$  *Ans.*
- $\Sigma M_y = 0; -C_x(0.4 \text{ m}) - 600 \text{ N}(0.6 \text{ m}) = 0$   
 $C_x = -900 \text{ N}$  *Ans.*
- $\Sigma M_x = 0; B_z(0.6 \text{ m}) + 600 \text{ N}(1.2 \text{ m})$   
 $+ (-400 \text{ N})(0.4 \text{ m}) = 0$   
 $B_z = -933.3 \text{ N}$  *Ans.*
- $\Sigma M_z = 0;$   
 $-B_x(0.6 \text{ m}) - (-900 \text{ N})(1.2 \text{ m})$   
 $+ (-400 \text{ N})(0.6 \text{ m}) = 0$   
 $B_x = 1400 \text{ N}$  *Ans.*
- $\Sigma F_x = 0; 1400 \text{ N} + (-900 \text{ N}) + A_x = 0$   
 $A_x = -500 \text{ N}$  *Ans.*
- $\Sigma F_z = 0; A_z - 933.3 \text{ N} + 600 \text{ N} = 0$   
 $A_z = 333.3 \text{ N}$  *Ans.*
- F5-10.**  $\Sigma F_x = 0; B_x = 0$  *Ans.*
- $\Sigma M_z = 0;$   
 $C_y(0.4 \text{ m} + 0.6 \text{ m}) = 0 \quad C_y = 0$  *Ans.*
- $\Sigma F_y = 0; A_y + 0 = 0 \quad A_y = 0$  *Ans.*
- $\Sigma M_x = 0; C_z(0.6 \text{ m} + 0.6 \text{ m}) + B_z(0.6 \text{ m})$   
 $- 450 \text{ N}(0.6 \text{ m} + 0.6 \text{ m}) = 0$   
 $1.2C_z + 0.6B_z - 540 = 0$   
 $\Sigma M_y = 0; -C_z(0.6 \text{ m} + 0.4 \text{ m})$   
 $- B_z(0.6 \text{ m}) + 450 \text{ N}(0.6 \text{ m}) = 0$   
 $-C_z - 0.6B_z + 270 = 0$   
 $C_z = 1350 \text{ N} \quad B_z = -1800 \text{ N}$  *Ans.*
- $\Sigma F_z = 0;$   
 $A_z + 1350 \text{ N} + (-1800 \text{ N}) - 450 \text{ N} = 0$   
 $A_z = 900 \text{ N}$  *Ans.*
- F5-11.**  $\Sigma F_y = 0; A_y = 0$  *Ans.*
- $\Sigma M_x = 0; -9(3) + F_{CE}(3) = 0$   
 $F_{CE} = 9 \text{ kN}$  *Ans.*
- $\Sigma M_z = 0; F_{CF}(3) - 6(3) = 0$   
 $F_{CF} = 6 \text{ kN}$  *Ans.*
- $\Sigma M_y = 0; 9(4) - A_z(4) - 6(1.5) = 0$   
 $A_z = 6.75 \text{ kN}$  *Ans.*
- $\Sigma F_x = 0; A_x + 6 - 6 = 0 \quad A_x = 0$  *Ans.*
- $\Sigma F_z = 0; F_{DB} + 9 - 9 + 6.75 = 0$   
 $F_{DB} = -6.75 \text{ kN}$  *Ans.*

- F5-12.**  $\Sigma F_x = 0; A_x = 0$  *Ans.*
- $\Sigma F_y = 0; A_y = 0$  *Ans.*
- $\Sigma F_z = 0; A_z + F_{BC} - 80 = 0$   
 $\Sigma M_x = 0; (M_A)_x + 6F_{BC} - 80(6) = 0$   
 $\Sigma M_y = 0; 3F_{BC} - 80(1.5) = 0 \quad F_{BC} = 40 \text{ lb}$  *Ans.*
- $\Sigma M_z = 0; (M_A)_z = 0$  *Ans.*
- $A_z = 40 \text{ lb} \quad (M_A)_x = 240 \text{ lb} \cdot \text{ft}$  *Ans.*

## Chapter 6

- F6-1.** *Joint A.*
- $\uparrow \Sigma F_y = 0; 225 \text{ lb} - F_{AD} \sin 45^\circ = 0$   
 $F_{AD} = 318.20 \text{ lb} = 318 \text{ lb (C)}$  *Ans.*
- $\rightarrow \Sigma F_x = 0; F_{AB} - (318.20 \text{ lb}) \cos 45^\circ = 0$   
 $F_{AB} = 225 \text{ lb (T)}$  *Ans.*
- Joint B.*
- $\rightarrow \Sigma F_x = 0; F_{BC} - 225 \text{ lb} = 0$   
 $F_{BC} = 225 \text{ lb (T)}$  *Ans.*
- $\uparrow \Sigma F_y = 0; F_{BD} = 0$  *Ans.*
- Joint D.*
- $\rightarrow \Sigma F_x = 0; F_{CD} \cos 45^\circ + (318.20 \text{ lb}) \cos 45^\circ - 450 \text{ lb} = 0$   
 $F_{CD} = 318.20 \text{ lb} = 318 \text{ lb (T)}$  *Ans.*
- F6-2.** *Joint D.*
- $\uparrow \Sigma F_y = 0; \frac{3}{5}F_{CD} - 300 = 0;$   
 $F_{CD} = 500 \text{ lb (T)}$  *Ans.*
- $\rightarrow \Sigma F_x = 0; -F_{AD} + \frac{4}{5}(500) = 0$   
 $F_{AD} = 400 \text{ lb (C)}$  *Ans.*
- $F_{BC} = 500 \text{ lb (T)}, F_{AC} = F_{AB} = 0$  *Ans.*
- F6-3.**  $D_x = 200 \text{ lb}, D_y = 650 \text{ lb}, B_y = 150 \text{ lb}$   
*Joint B.*
- $\rightarrow \Sigma F_x = 0; F_{BA} = 0$  *Ans.*
- $\uparrow \Sigma F_y = 0; 150 - F_{BC} = 0; F_{BC} = 150 \text{ lb (C)}$  *Ans.*
- Joint A.*
- $\rightarrow \Sigma F_x = 0; F_{AC}\left(\frac{4}{5}\right) = 0; F_{AC} = 0$  *Ans.*
- $\uparrow \Sigma F_y = 0; F_{AD} - 800 = 0; F_{AD} = 800 \text{ lb (T)}$  *Ans.*
- Joint C.*
- $\rightarrow \Sigma F_x = 0; -F_{CD} + 200 = 0; F_{CD} = 200 \text{ lb (T)}$  *Ans.*

**F6-4.** Joint C.

$$+\uparrow \sum F_y = 0; \quad 2F \cos 30^\circ - P = 0 \\ F_{AC} = F_{BC} = F = \frac{P}{2 \cos 30^\circ} = 0.5774P \text{ (C)}$$

Joint B.

$$+\rightarrow \sum F_x = 0; 0.5774P \cos 60^\circ - F_{AB} = 0 \\ F_{AB} = 0.2887P \text{ (T)} \\ F_{AB} = 0.2887P = 2 \text{ kN} \\ P = 6.928 \text{ kN} \\ F_{AC} = F_{BC} = 0.5774P = 1.5 \text{ kN} \\ P = 2.598 \text{ kN}$$

The smaller value of  $P$  is chosen,

$$P = 2.598 \text{ kN} = 2.60 \text{ kN}$$

**F6-8.**  $\zeta + \sum M_A = 0; \quad G_y(12 \text{ m}) - 20 \text{ kN}(2 \text{ m})$

$$- 30 \text{ kN}(4 \text{ m}) - 40 \text{ kN}(6 \text{ m}) = 0 \\ G_y = 33.33 \text{ kN}$$

$$+\uparrow \sum F_y = 0; F_{KC} + 33.33 \text{ kN} - 40 \text{ kN} = 0 \\ F_{KC} = 6.67 \text{ kN} \text{ (C)}$$

**Ans.**

$$\zeta + \sum M_K = 0; \\ 33.33 \text{ kN}(8 \text{ m}) - 40 \text{ kN}(2 \text{ m}) - F_{CD}(3 \text{ m}) = 0 \\ F_{CD} = 62.22 \text{ kN} = 62.2 \text{ kN} \text{ (T)}$$

**Ans.**

$$+\rightarrow \sum F_x = 0; F_{LK} - 62.22 \text{ kN} = 0 \\ F_{LK} = 62.2 \text{ kN} \text{ (C)}$$

**Ans.****F6-5.**  $F_{CB} = 0$ **Ans.**

$$F_{CD} = 0$$

**Ans.**

$$F_{AE} = 0$$

**Ans.**

$$F_{DE} = 0$$

**Ans.****F6-6.** Joint C.

$$+\uparrow \sum F_y = 0; 259.81 \text{ lb} - F_{CD} \sin 30^\circ = 0 \\ F_{CD} = 519.62 \text{ lb} = 520 \text{ lb} \text{ (C)}$$

**Ans.**

$$+\rightarrow \sum F_x = 0; (519.62 \text{ lb}) \cos 30^\circ - F_{BC} = 0$$

**Ans.**

$$F_{BC} = 450 \text{ lb} \text{ (T)}$$

Joint D.

$$+\nearrow \sum F_{y'} = 0; F_{BD} \cos 30^\circ = 0 \quad F_{BD} = 0$$

**Ans.**

$$+\searrow \sum F_{x'} = 0; F_{DE} - 519.62 \text{ lb} = 0$$

**Ans.**

$$F_{DE} = 519.62 \text{ lb} = 520 \text{ lb} \text{ (C)}$$

Joint B.

$$\uparrow \sum F_y = 0; F_{BE} \sin \phi = 0 \quad F_{BE} = 0$$

**Ans.**

$$+\rightarrow \sum F_x = 0; 450 \text{ lb} - F_{AB} = 0$$

**Ans.**

$$F_{AB} = 450 \text{ lb} \text{ (T)}$$

Joint A.

$$+\uparrow \sum F_y = 0; 340.19 \text{ lb} - F_{AE} = 0$$

**Ans.**

$$F_{AE} = 340 \text{ lb} \text{ (C)}$$

**F6-7.**  $+\uparrow \sum F_y = 0; F_{CF} \sin 45^\circ - 600 - 800 = 0$ 

$$F_{CF} = 1980 \text{ lb} \text{ (T)}$$

**Ans.**

$$\zeta + \sum M_C = 0; F_{FE}(4) - 800(4) = 0$$

$$F_{FE} = 800 \text{ lb} \text{ (T)}$$

**Ans.**

$$\zeta + \sum M_F = 0; F_{BC}(4) - 600(4) - 800(8) = 0$$

$$F_{BC} = 2200 \text{ lb} \text{ (C)}$$

**Ans.**

**F6-8.**  $\zeta + \sum M_A = 0; \quad G_y(12 \text{ m}) - 20 \text{ kN}(2 \text{ m})$

$$- 30 \text{ kN}(4 \text{ m}) - 40 \text{ kN}(6 \text{ m}) = 0 \\ G_y = 33.33 \text{ kN}$$

$$+\uparrow \sum F_y = 0; F_{KC} + 33.33 \text{ kN} - 40 \text{ kN} = 0 \\ F_{KC} = 6.67 \text{ kN} \text{ (C)}$$

**Ans.**

$$\zeta + \sum M_K = 0; \\ 33.33 \text{ kN}(8 \text{ m}) - 40 \text{ kN}(2 \text{ m}) - F_{CD}(3 \text{ m}) = 0$$

**Ans.**

$$+\rightarrow \sum F_x = 0; F_{LK} - 62.22 \text{ kN} = 0 \\ F_{LK} = 62.2 \text{ kN} \text{ (C)}$$

**Ans.**

From the geometry of the truss,

$$\phi = \tan^{-1}(3 \text{ m}/2 \text{ m}) = 56.31^\circ.$$

$$\zeta + \sum M_K = 0;$$

$$33.33 \text{ kN}(8 \text{ m}) - 40 \text{ kN}(2 \text{ m}) - F_{CD}(3 \text{ m}) = 0$$

**Ans.**

$$F_{CD} = 62.2 \text{ kN} \text{ (T)}$$

**Ans.**

$$\zeta + \sum M_D = 0; 33.33 \text{ kN}(6 \text{ m}) - F_{KJ}(3 \text{ m}) = 0$$

$$F_{KJ} = 66.7 \text{ kN} \text{ (C)}$$

**Ans.**

$$+\uparrow \sum F_y = 0;$$

$$33.33 \text{ kN} - 40 \text{ kN} + F_{KD} \sin 56.31^\circ = 0$$

$$F_{KD} = 8.01 \text{ kN} \text{ (T)}$$

**Ans.**

From the geometry of the truss,

$$\tan \phi = \frac{(9 \text{ ft}) \tan 30^\circ}{3 \text{ ft}} = 1.732 \quad \phi = 60^\circ$$

$$\zeta + \sum M_C = 0;$$

$$F_{EF} \sin 30^\circ(6 \text{ ft}) + 300 \text{ lb}(6 \text{ ft}) = 0$$

$$F_{EF} = -600 \text{ lb} = 600 \text{ lb} \text{ (C)}$$

**Ans.**

$$\zeta + \sum M_D = 0;$$

$$300 \text{ lb}(6 \text{ ft}) - F_{CF} \sin 60^\circ(6 \text{ ft}) = 0$$

$$F_{CF} = 346.41 \text{ lb} = 346 \text{ lb} \text{ (T)}$$

**Ans.**

$$\zeta + \sum M_F = 0;$$

$$300 \text{ lb}(9 \text{ ft}) - 300 \text{ lb}(3 \text{ ft}) - F_{BC}(9 \text{ ft}) \tan 30^\circ = 0$$

$$F_{BC} = 346.41 \text{ lb} = 346 \text{ lb} \text{ (T)}$$

**Ans.**

From the geometry of the truss,

$$\theta = \tan^{-1}(1 \text{ m}/2 \text{ m}) = 26.57^\circ$$

$$\phi = \tan^{-1}(3 \text{ m}/2 \text{ m}) = 56.31^\circ.$$

The location of  $O$  can be found using similar triangles.

$$\frac{1 \text{ m}}{2 \text{ m}} = \frac{2 \text{ m}}{2 \text{ m} + x}$$

$$4 \text{ m} = 2 \text{ m} + x$$

$$x = 2 \text{ m}$$

$$\zeta + \sum M_G = 0;$$

$$26.25 \text{ kN}(4 \text{ m}) - 15 \text{ kN}(2 \text{ m}) - F_{CD}(3 \text{ m}) = 0$$

$$F_{CD} = 25 \text{ kN (T)}$$

*Ans.*

$$\zeta + \sum M_D = 0;$$

$$26.25 \text{ kN}(2 \text{ m}) - F_{GF} \cos 26.57^\circ(2 \text{ m}) = 0$$

$$F_{GF} = 29.3 \text{ kN (C)}$$

*Ans.*

$$\zeta + \sum M_O = 0; 15 \text{ kN}(4 \text{ m}) - 26.25 \text{ kN}(2 \text{ m})$$

$$- F_{GD} \sin 56.31^\circ(4 \text{ m}) = 0$$

$$F_{GD} = 2.253 \text{ kN} = 2.25 \text{ kN (T)}$$

*Ans.*

**F6-12.**  $\zeta + \sum M_H = 0;$

$$F_{DC}(12 \text{ ft}) + 1200 \text{ lb}(9 \text{ ft}) - 1600 \text{ lb}(21 \text{ ft}) = 0$$

$$F_{DC} = 1900 \text{ lb (C)}$$

*Ans.*

$$\zeta + \sum M_D = 0;$$

$$1200 \text{ lb}(21 \text{ ft}) - 1600 \text{ lb}(9 \text{ ft}) - F_{HI}(12 \text{ ft}) = 0$$

$$F_{HI} = 900 \text{ lb (C)}$$

*Ans.*

$$\zeta + \sum M_C = 0; F_{JI} \cos 45^\circ(12 \text{ ft}) + 1200 \text{ lb}(21 \text{ ft})$$

$$- 900 \text{ lb}(12 \text{ ft}) - 1600 \text{ lb}(9 \text{ ft}) = 0$$

$$F_{JI} = 0$$

*Ans.*

**F6-13.**  $+ \uparrow \sum F_y = 0; 3P - 60 = 0$

$$P = 20 \text{ lb}$$

*Ans.*

**F6-14.**  $\zeta + \sum M_C = 0;$

$$-\left(\frac{4}{5}\right)(F_{AB})(9) + 400(6) + 500(3) = 0$$

$$F_{AB} = 541.67 \text{ lb}$$

$$+ \rightarrow \sum F_x = 0; -C_x + \frac{3}{5}(541.67) = 0$$

$$C_x = 325 \text{ lb}$$

*Ans.*

$$+ \uparrow \sum F_y = 0; C_y + \frac{4}{5}(541.67) - 400 - 500 = 0$$

$$C_y = 467 \text{ lb}$$

*Ans.*

**F6-15.**  $\zeta + \sum M_A = 0; 100 \text{ N}(250 \text{ mm}) - N_B(50 \text{ mm}) = 0$

$$N_B = 500 \text{ N}$$

*Ans.*

$$+ \rightarrow \sum F_x = 0; (500 \text{ N}) \sin 45^\circ - A_x = 0$$

$$A_x = 353.55 \text{ N}$$

$$+ \uparrow \sum F_y = 0; A_y - 100 \text{ N} - (500 \text{ N}) \cos 45^\circ = 0$$

$$A_y = 453.55 \text{ N}$$

$$F_A = \sqrt{(353.55 \text{ N})^2 + (453.55 \text{ N})^2}$$

$$= 575 \text{ N}$$

*Ans.*

**F6-16.**  $\zeta + \sum M_C = 0;$

$$400(2) + 800 - F_{BA} \left(\frac{3}{\sqrt{10}}\right)(1)$$

$$- F_{BA} \left(\frac{1}{\sqrt{10}}\right)(3) = 0$$

$$F_{BA} = 843.27 \text{ N}$$

$$+ \rightarrow \sum F_x = 0; C_x - 843.27 \left(\frac{3}{\sqrt{10}}\right) = 0$$

$$C_x = 800 \text{ N}$$

$$+ \uparrow \sum F_y = 0; C_y + 843.27 \left(\frac{1}{\sqrt{10}}\right) - 400 = 0$$

$$C_y = 133 \text{ N}$$

*Ans.*

**F6-17.** Plate *A*:

$$+ \uparrow \sum F_y = 0; 2T + N_{AB} - 100 = 0$$

Plate *B*:

$$+ \uparrow \sum F_y = 0; 2T - N_{AB} - 30 = 0$$

$$T = 32.5 \text{ lb}, N_{AB} = 35 \text{ lb}$$

*Ans.*

**F6-18.** Pulley *C*:

$$+ \uparrow \sum F_y = 0; T - 2P = 0; T = 2P$$

Beam:

$$+ \uparrow \sum F_y = 0; 2P + P - 6 = 0$$

$$P = 2 \text{ kN}$$

$$\zeta + \sum M_A = 0; 2(1) - 6(x) = 0$$

$$x = 0.333 \text{ m}$$

*Ans.*

**F6-19.** Member *CD*

$$\zeta + \sum M_D = 0; 600(1.5) - N_C(3) = 0$$

$$N_C = 300 \text{ N}$$

Member *ABC*

$$\zeta + \sum M_A = 0; -800 + B_y(2) - (300 \sin 45^\circ)4 = 0$$

$$B_y = 824.26 = 824 \text{ N}$$

*Ans.*

$$+ \rightarrow \sum F_x = 0; A_x - 300 \cos 45^\circ = 0;$$

$$A_x = 212 \text{ N}$$

*Ans.*

$$+ \uparrow \sum F_y = 0; -A_y + 824.26 - 300 \sin 45^\circ = 0;$$

$$A_y = 612 \text{ N}$$

*Ans.*

**F6-20.** *AB* is a two-force member.

Member *BC*

$$\zeta + \sum M_c = 0; 15(3) + 10(6) - F_{BC}\left(\frac{4}{5}\right)(9) = 0$$

$$F_{BC} = 14.58 \text{ kN}$$

$$+ \rightarrow \sum F_x = 0; (14.58)\left(\frac{3}{5}\right) - C_x = 0;$$

$$C_x = 8.75 \text{ kN}$$

$$+ \uparrow \sum F_y = 0; (14.58)\left(\frac{4}{5}\right) - 10 - 15 + C_y = 0;$$

$$C_y = 13.3 \text{ kN}$$

Member *CD*

$$+ \rightarrow \sum F_x = 0; 8.75 - D_x = 0; D_x = 8.75 \text{ kN}$$

*Ans.*

$$+ \uparrow \sum F_y = 0; -13.3 + D_y = 0; D_y = 13.3 \text{ kN}$$

*Ans.*

$$\zeta + \sum M_D = 0; -8.75(4) + M_D = 0; M_D = 35 \text{ kN} \cdot \text{m}$$

*Ans.*

**F6-21.** Entire frame

$$\begin{aligned}\zeta + \sum M_A &= 0; -600(3) - [400(3)](1.5) + C_y(3) = 0 \\ C_y &= 1200 \text{ N} \quad \text{Ans.} \\ +\uparrow \sum F_y &= 0; A_y - 400(3) + 1200 = 0 \\ A_y &= 0 \quad \text{Ans.} \\ +\rightarrow \sum F_x &= 0; 600 - A_x - C_x = 0 \\ \text{Member } AB \\ \zeta + \sum M_B &= 0; 400(1.5)(0.75) - A_x(3) = 0 \\ A_x &= 150 \text{ N} \quad \text{Ans.} \\ C_x &= 450 \text{ N} \quad \text{Ans.}\end{aligned}$$

These same results can be obtained by considering members *AB* and *BC*.

**F6-22.** Entire frame

$$\begin{aligned}\zeta + \sum M_E &= 0; 250(6) - A_y(6) = 0 \\ A_y &= 250 \text{ N} \\ +\rightarrow \sum F_x &= 0; E_x = 0 \\ +\uparrow \sum F_y &= 0; 250 - 250 + E_y = 0; E_y = 0 \\ \text{Member } BD \\ \zeta + \sum M_D &= 0; 250(4.5) - B_y(3) = 0; \\ B_y &= 375 \text{ N} \\ \text{Member } ABC \\ \zeta + \sum M_C &= 0; -250(3) + 375(1.5) + B_x(2) = 0 \\ B_x &= 93.75 \text{ N} \\ +\rightarrow \sum F_x &= 0; C_x - B_x = 0; C_x = 93.75 \text{ N} \quad \text{Ans.} \\ +\uparrow \sum F_y &= 0; 250 - 375 + C_y = 0; C_y = 125 \text{ N} \quad \text{Ans.}\end{aligned}$$

**F6-23.** *AD, CB* are two-force members.Member *AB*

$$\begin{aligned}\zeta + \sum M_A &= 0; -[\frac{1}{2}(3)(4)](1.5) + B_y(3) = 0 \\ B_y &= 3 \text{ kN}\end{aligned}$$

Since *BC* is a two-force member  $C_y = B_y = 3 \text{ kN}$  and  $C_x = 0$  ( $\sum M_B = 0$ ).

Member *EDC*

$$\begin{aligned}\zeta + \sum M_E &= 0; F_{DA}(\frac{4}{5})(1.5) - 5(3) - 3(3) = 0 \\ F_{DA} &= 20 \text{ kN} \\ +\rightarrow \sum F_x &= 0; E_x - 20(\frac{3}{5}) = 0; E_x = 12 \text{ kN} \quad \text{Ans.} \\ +\uparrow \sum F_y &= 0; -E_y + 20(\frac{4}{5}) - 5 - 3 = 0; \\ E_y &= 8 \text{ kN} \quad \text{Ans.}\end{aligned}$$

**F6-24.** *AC* and *DC* are two-force members.

$$\begin{aligned}\text{Member } BC \\ \zeta + \sum M_C &= 0; [\frac{1}{2}(3)(8)](1) - B_y(3) = 0 \\ B_y &= 4 \text{ kN} \\ \text{Member } BA \\ \zeta + \sum M_B &= 0; 6(2) - A_x(4) = 0 \\ A_x &= 3 \text{ kN} \quad \text{Ans.} \\ +\uparrow \sum F_y &= 0; -4 \text{ kN} + A_y = 0; A_y = 4 \text{ kN} \quad \text{Ans.} \\ \text{Entire Frame} \\ \zeta + \sum M_A &= 0; -6(2) - [\frac{1}{2}(3)(8)](2) + D_y(3) = 0 \\ D_y &= 12 \text{ kN} \quad \text{Ans.} \\ \text{Since } DC \text{ is a two-force member } (\sum M_C = 0) \text{ then} \\ D_x &= 0 \quad \text{Ans.}\end{aligned}$$

**Chapter 7**

$$\begin{aligned}\text{F7-1.} \quad \zeta + \sum M_A &= 0; B_y(6) - 10(1.5) - 15(4.5) = 0 \\ B_y &= 13.75 \text{ kN} \\ +\rightarrow \sum F_x &= 0; N_C = 0 \quad \text{Ans.} \\ +\uparrow \sum F_y &= 0; V_C + 13.75 - 15 = 0 \\ V_C &= 1.25 \text{ kN} \quad \text{Ans.} \\ \zeta + \sum M_C &= 0; 13.75(3) - 15(1.5) - M_C = 0 \\ M_C &= 18.75 \text{ kN}\cdot\text{m} \quad \text{Ans.} \\ \text{F7-2.} \quad \zeta + \sum M_B &= 0; 30 - 10(1.5) - A_y(3) = 0 \\ A_y &= 5 \text{ kN} \\ +\rightarrow \sum F_x &= 0; N_C = 0 \quad \text{Ans.} \\ +\uparrow \sum F_y &= 0; 5 - V_C = 0 \\ V_C &= 5 \text{ kN} \quad \text{Ans.} \\ \zeta + \sum M_C &= 0; M_C + 30 - 5(1.5) = 0 \\ M_C &= -22.5 \text{ kN}\cdot\text{m} \quad \text{Ans.} \\ \text{F7-3.} \quad +\rightarrow \sum F_x &= 0; B_x = 0 \\ \zeta + \sum M_A &= 0; 3(6)(3) - B_y(9) = 0 \\ B_y &= 6 \text{ kip} \\ +\rightarrow \sum F_x &= 0; N_C = 0 \quad \text{Ans.} \\ \zeta + \uparrow \sum F_y &= 0; V_C - 6 = 0 \\ V_C &= 6 \text{ kip} \quad \text{Ans.} \\ \zeta + \sum M_C &= 0; -M_C - 6(4.5) = 0 \\ M_C &= -27 \text{ kip}\cdot\text{ft} \quad \text{Ans.} \\ \text{F7-4.} \quad \zeta + \sum M_A &= 0; B_y(6) - 12(1.5) - 9(3)(4.5) = 0 \\ B_y &= 23.25 \text{ kN} \\ +\rightarrow \sum F_x &= 0; N_C = 0 \quad \text{Ans.} \\ +\uparrow \sum F_y &= 0; V_C + 23.25 - 9(1.5) = 0 \\ V_C &= -9.75 \text{ kN} \quad \text{Ans.} \\ \zeta + \sum M_C &= 0; \\ 23.25(1.5) - 9(1.5)(0.75) - M_C &= 0 \\ M_C &= 24.75 \text{ kN}\cdot\text{m} \quad \text{Ans.}\end{aligned}$$

**F7-5.**  $\zeta + \sum M_A = 0; B_y(6) - \frac{1}{2}(9)(6)(3) = 0$   
 $B_y = 13.5 \text{ kN}$

$\stackrel{+}{\rightarrow} \sum F_x = 0; N_C = 0$

$\stackrel{+}{\uparrow} \sum F_y = 0; V_C + 13.5 - \frac{1}{2}(9)(3) = 0$

$V_C = 0$

$\zeta + \sum M_C = 0; 13.5(3) - \frac{1}{2}(9)(1) - M_C = 0$   
 $M_C = 27 \text{ kN}\cdot\text{m}$

*Ans.*  
*Ans.*

**F7-6.**  $\zeta + \sum M_A = 0;$   
 $B_y(6) - \frac{1}{2}(6)(3)(2) - 6(3)(4.5) = 0$   
 $B_y = 16.5 \text{ kN}$

$\stackrel{+}{\rightarrow} \sum F_x = 0; N_C = 0$

$\stackrel{+}{\uparrow} \sum F_y = 0; V_C + 16.5 - 6(3) = 0$

$V_C = 1.50 \text{ kN}$

$\zeta + \sum M_C = 0; 16.5(3) - 6(3)(1.5) - M_C = 0$   
 $M_C = 22.5 \text{ kN}\cdot\text{m}$

*Ans.*  
*Ans.*

**F7-7.**  $\stackrel{+}{\uparrow} \sum F_y = 0; 6 - V = 0 \quad V = 6 \text{ kN}$   
 $\zeta + \sum M_O = 0; M + 18 - 6x = 0$   
 $M = (6x - 18) \text{ kN}\cdot\text{m}$

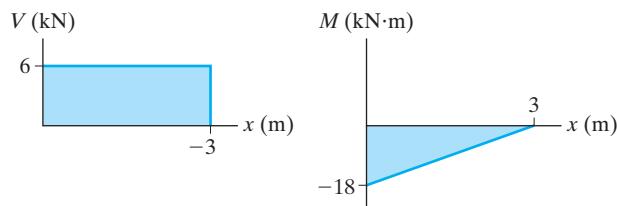


Fig. F7-7

**F7-8.**  $\stackrel{+}{\uparrow} \sum F_y = 0; -V - 2x = 0$   
 $V = (-2x) \text{ kN}$   
 $\zeta + \sum M_O = 0; M + 2x(\frac{x}{2}) - 15 = 0$   
 $M = (15 - x^2) \text{ kN}\cdot\text{m}$

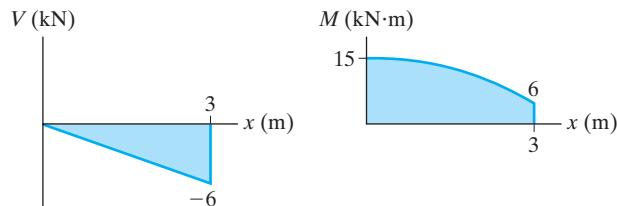


Fig. F7-8

**F7-9.**  $\stackrel{+}{\uparrow} \sum F_y = 0; -V - \frac{1}{2}(2x)(x) = 0$   
 $V = -(x^2) \text{ kN}$

$\zeta + \sum M_O = 0; M + \frac{1}{2}(2x)(x)(\frac{x}{3}) = 0$   
 $M = -(\frac{1}{3}x^3) \text{ kN}\cdot\text{m}$

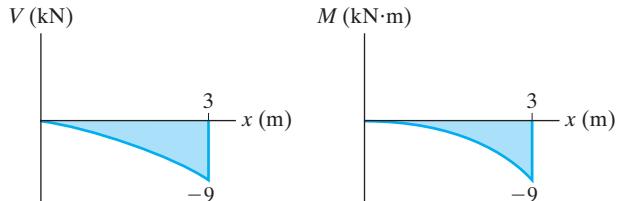


Fig. F7-9

**F7-10.**  $\stackrel{+}{\uparrow} \sum F_y = 0; -V - 2x = 0$   
 $V = -2 \text{ kN}$

$\zeta + \sum M_O = 0; M + 2x = 0$   
 $M = (-2x) \text{ kN}\cdot\text{m}$

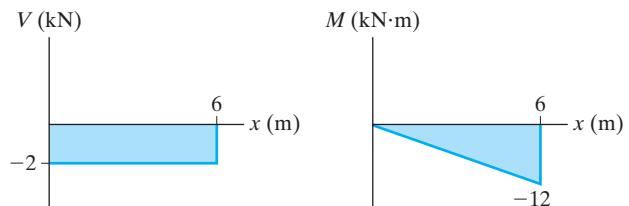


Fig. F7-10

**F7-11.** Region  $3 \leq x < 3 \text{ m}$   
 $\stackrel{+}{\uparrow} \sum F_y = 0; -V - 5 = 0 \quad V = -5 \text{ kN}$   
 $\zeta + \sum M_O = 0; M + 5x = 0$   
 $M = (-5x) \text{ kN}\cdot\text{m}$

Region  $0 < x \leq 6 \text{ m}$

$\stackrel{+}{\uparrow} \sum F_y = 0; V + 5 = 0 \quad V = -5 \text{ kN}$

$\zeta + \sum M_O = 0; 5(6 - x) - M = 0$

$M = (5(6 - x)) \text{ kN}\cdot\text{m}$

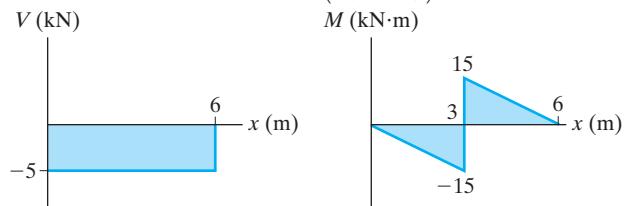


Fig. F7-11

**F7-12.** Region  $0 \leq x < 3$  m  
 $\uparrow \sum F_y = 0; V = 0$   
 $\zeta + \sum M_O = 0; M - 12 = 0$   
 $M = 12 \text{ kN}\cdot\text{m}$

Region  $3 \leq x \leq 6$  m  
 $\uparrow \sum F_y = 0; V + 4 = 0 \quad V = -4 \text{ kN}$   
 $\zeta + \sum M_O = 0; 4(6 - x) - M = 0$   
 $M = (4(6 - x)) \text{ kN}\cdot\text{m}$

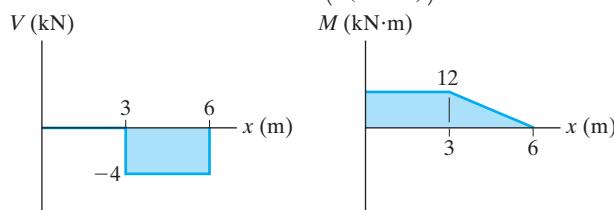


Fig. F7-12

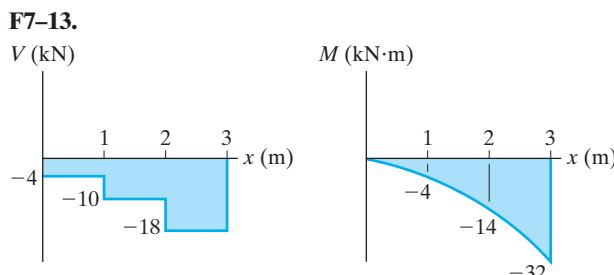


Fig. F7-13

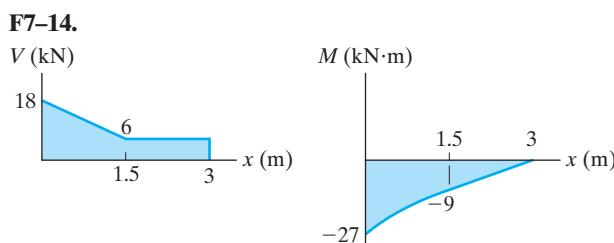


Fig. F7-14

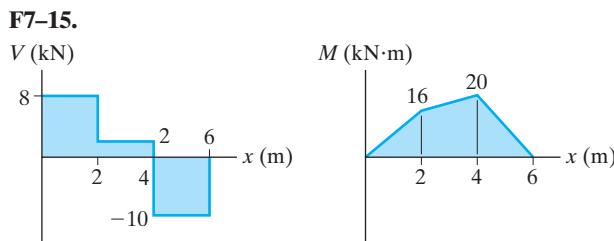


Fig. F7-15

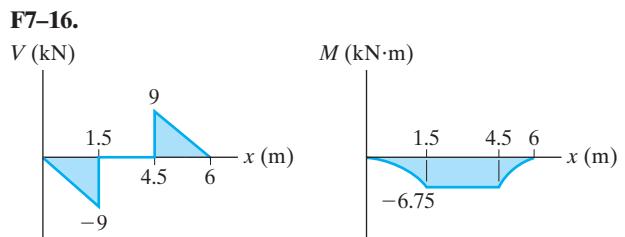


Fig. F7-16

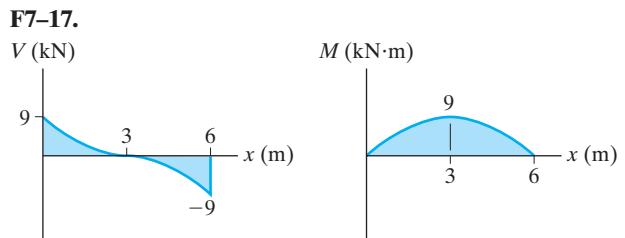


Fig. F7-17

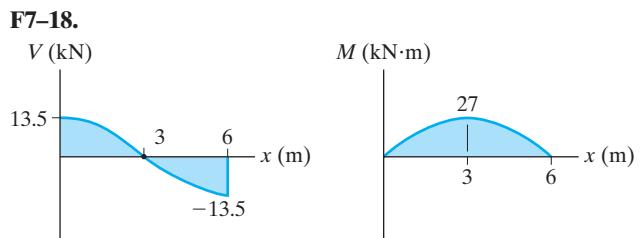


Fig. F7-18

## Chapter 8

**F8-1.** a)  $\uparrow \sum F_y = 0; N - 50(9.81) - 200\left(\frac{3}{5}\right) = 0$   
 $N = 610.5 \text{ N}$

$\rightarrow \sum F_x = 0; F - 200\left(\frac{4}{5}\right) = 0$   
 $F = 160 \text{ N}$

$F < F_{\max} = \mu_s N = 0.3(610.5) = 183.15 \text{ N}$ ,  
therefore  $F = 160 \text{ N}$

*Ans.*

b)  $\uparrow \sum F_y = 0; N - 50(9.81) - 400\left(\frac{3}{5}\right) = 0$   
 $N = 730.5 \text{ N}$

$\rightarrow \sum F_x = 0; F - 400\left(\frac{4}{5}\right) = 0$   
 $F = 320 \text{ N}$

$F > F_{\max} = \mu_s N = 0.3(730.5) = 219.15 \text{ N}$

Block slips

$F = \mu_s N = 0.2(730.5) = 146 \text{ N}$

*Ans.*

**F8-2.**  $\zeta + \sum M_B = 0;$

$$N_A(3) + 0.2N_A(4) - 30(9.81)(2) = 0$$

$$N_A = 154.89 \text{ N}$$

$$\stackrel{+}{\rightarrow} \sum F_x = 0; \quad P - 154.89 = 0$$

$$P = 154.89 \text{ N} = 155 \text{ N}$$

*Ans.*

**F8-3.** Crate *A*

$$+\uparrow \sum F_y = 0; \quad N_A - 50(9.81) = 0$$

$$N_A = 490.5 \text{ N}$$

$$\stackrel{+}{\rightarrow} \sum F_x = 0; \quad T - 0.25(490.5) = 0$$

$$T = 122.62 \text{ N}$$

Crate *B*

$$+\uparrow \sum F_y = 0; \quad N_B + P \sin 30^\circ - 50(9.81) = 0$$

$$N_B = 490.5 - 0.5P$$

$$\stackrel{+}{\rightarrow} \sum F_x = 0;$$

$$P \cos 30^\circ - 0.25(490.5 - 0.5P) = 122.62 = 0$$

$$P = 247 \text{ N}$$

*Ans.*

**F8-4.**  $\stackrel{+}{\rightarrow} \sum F_x = 0; \quad N_A - 0.3N_B = 0$

$$+\uparrow \sum F_y = 0;$$

$$N_B + 0.3N_A + P - 100(9.81) = 0$$

$$\zeta + \sum M_O = 0;$$

$$P(0.6) - 0.3N_B(0.9) - 0.3N_A(0.9) = 0$$

$$N_A = 175.70 \text{ N} \quad N_B = 585.67 \text{ N}$$

$$P = 343 \text{ N}$$

*Ans.*

**F8-5.** If slipping occurs:

$$+\uparrow \sum F_y = 0; \quad N_c - 250 \text{ lb} = 0; \quad N_c = 250 \text{ lb}$$

$$\stackrel{+}{\rightarrow} \sum F_x = 0; \quad P - 0.4(250) = 0; \quad P = 100 \text{ lb}$$

If tipping occurs:

$$\zeta + \sum M_A = 0; \quad -P(4.5) + 250(1.5) = 0$$

$$P = 83.3 \text{ lb}$$

*Ans.*

**F8-6.**

$$\zeta + \sum M_A = 0; \quad 490.5(0.6) - T \cos 60^\circ (0.3 \cos 60^\circ + 0.6) - T \sin 60^\circ (0.3 \sin 60^\circ) = 0$$

$$T = 490.5 \text{ N}$$

$$\stackrel{+}{\rightarrow} \sum F_x = 0; \quad 490.5 \sin 60^\circ - N_A = 0; \quad N_A = 424.8 \text{ N}$$

$$+\uparrow \sum F_y = 0; \quad \mu_s(424.8) + 490.5 \cos 60^\circ - 490.5 = 0$$

$$\mu_s = 0.577$$

*Ans.*

**F8-7.**

*A* will not move. Assume *B* is about to slip on *C* and *A*, and *C* is stationary.

$$\stackrel{+}{\rightarrow} \sum F_x = 0; \quad P - 0.3(50) - 0.4(75) = P = 45 \text{ N}$$

Assume *C* is about to slip and *B* does not slip on *C*, but is about to slip at *A*.

$$\stackrel{+}{\rightarrow} \sum F_x = 0; \quad P - 0.3(50) - 0.35(90) = 0$$

$$P = 46.5 \text{ N} > 45 \text{ N}$$

$$P = 45 \text{ N} \quad \text{Ans.}$$

**F8-8.**

*A* is about to move down the plane and *B* moves upward.

Block *A*

$$+\nwarrow \sum F_y = 0; \quad N = W \cos \theta$$

$$+\nearrow \sum F_x = 0; \quad T + \mu_s(W \cos \theta) - W \sin \theta = 0$$

$$T = W \sin \theta - \mu_s W \cos \theta \quad (1)$$

Block *B*

$$+\nwarrow \sum F_y = 0; \quad N' = 2W \cos \theta$$

$$+\nearrow \sum F_x = 0; \quad 2T - \mu_s W \cos \theta - \mu_s(2W \cos \theta) - W \sin \theta = 0$$

Using Eq.(1);

$$\theta = \tan^{-1} 5\mu_s$$

*Ans.*

**F8-9.**

Assume *B* is about to slip on *A*,  $F_B = 0.3 N_B$ .

$$\stackrel{+}{\rightarrow} \sum F_x = 0; \quad P - 0.3(10)(9.81) = 0$$

$$P = 29.4 \text{ N}$$

Assume *B* is about to tip on *A*,  $x = 0$ .

$$\zeta + \sum M_O = 0; \quad 10(9.81)(0.15) - P(0.4) = 0$$

$$P = 36.8 \text{ N}$$

Assume *A* is about to slip,  $F_A = 0.1 N_A$ .

$$\stackrel{+}{\rightarrow} \sum F_x = 0 \quad P - 0.1[7(9.81) + 10(9.81)] = 0$$

$$P = 16.7 \text{ N}$$

Choose the smallest result.  $P = 16.7 \text{ N} \quad \text{Ans.}$

## Chapter 9

$$\mathbf{F9-1.} \quad \bar{x} = \frac{\int_A \tilde{x} dA}{\int_A dA} = \frac{\frac{1}{2} \int_0^1 y^{2/3} dy}{\int_0^1 y^{1/3} dy} = 0.4 \text{ m} \quad \text{Ans.}$$

$$\bar{y} = \frac{\int_A \tilde{y} dA}{\int_A dA} = \frac{\int_0^1 y^{4/3} dy}{\int_0^1 y^{1/3} dy} = 0.571 \text{ m} \quad \text{Ans.}$$

$$\mathbf{F9-2.} \quad \bar{x} = \frac{\int_A \tilde{x} dA}{\int_A dA} = \frac{\int_0^1 x(x^3 dx)}{\int_0^1 x^3 dx}$$

$$= 0.8 \text{ m}$$

$$\bar{y} = \frac{\int_A \tilde{y} dA}{\int_A dA} = \frac{\int_0^1 \frac{1}{2} x^3 (x^3 dx)}{\int_0^1 x^3 dx}$$

$$= 0.286 \text{ m}$$

*Ans.*

$$\mathbf{F9-3.} \quad \bar{y} = \frac{\int_A \tilde{y} dA}{\int_A dA} = \frac{\int_0^{2 \text{ m}} y \left( 2 \left( \frac{y^{1/2}}{\sqrt{2}} \right) \right) dy}{\int_0^{2 \text{ m}} 2 \left( \frac{y^{1/2}}{\sqrt{2}} \right) dy}$$

$$= 1.2 \text{ m}$$

*Ans.*

$$\mathbf{F9-4.} \quad \bar{x} = \frac{\int_m \tilde{x} dm}{\int_m dm} = \frac{\int_0^L x \left[ m_0 \left( 1 + \frac{x^2}{L^2} \right) dx \right]}{\int_0^L m_0 \left( 1 + \frac{x^2}{L^2} \right) dx}$$

$$= \frac{9}{16} L$$

*Ans.*

$$\mathbf{F9-5.} \quad \bar{y} = \frac{\int_V \tilde{y} dV}{\int_V dV} = \frac{\int_0^{1 \text{ m}} y \left( \frac{\pi}{4} y dy \right)}{\int_0^{1 \text{ m}} \frac{\pi}{4} y dy}$$

$$= 0.667 \text{ m}$$

*Ans.*

$$\mathbf{F9-6.} \quad \bar{z} = \frac{\int_V \tilde{z} dV}{\int_V dV} = \frac{\int_0^2 z \left[ \frac{9\pi}{64} (4-z)^2 dz \right]}{\int_0^2 \frac{9\pi}{64} (4-z)^2 dz}$$

$$= 0.786 \text{ ft}$$

*Ans.*

$$\mathbf{F9-7.} \quad \bar{x} = \frac{\Sigma \tilde{x} L}{\Sigma L} = \frac{150(300) + 300(600) + 300(400)}{300 + 600 + 400}$$

$$= 265 \text{ mm}$$

*Ans.*

$$\bar{y} = \frac{\Sigma \tilde{y} L}{\Sigma L} = \frac{0(300) + 300(600) + 600(400)}{300 + 600 + 400}$$

$$= 323 \text{ mm}$$

*Ans.*

$$\bar{z} = \frac{\Sigma \tilde{z} L}{\Sigma L} = \frac{0(300) + 0(600) + (-200)(400)}{300 + 600 + 400}$$

$$= -61.5 \text{ mm}$$

*Ans.*

$$\mathbf{F9-8.} \quad \bar{y} = \frac{\Sigma \tilde{y} A}{\Sigma A} = \frac{150[300(50)] + 325[50(300)]}{300(50) + 50(300)}$$

$$= 237.5 \text{ mm}$$

*Ans.*

$$\mathbf{F9-9.} \quad \bar{y} = \frac{\Sigma \tilde{y} A}{\Sigma A} = \frac{100[2(200)(50)] + 225[50(400)]}{2(200)(50) + 50(400)}$$

$$= 162.5 \text{ mm}$$

*Ans.*

$$\mathbf{F9-10.} \quad \bar{x} = \frac{\Sigma \tilde{x} A}{\Sigma A} = \frac{0.25[4(0.5)] + 1.75[0.5(2.5)]}{4(0.5) + 0.5(2.5)}$$

$$= 0.827 \text{ in.}$$

*Ans.*

$$\bar{y} = \frac{\Sigma \tilde{y} A}{\Sigma A} = \frac{2[4(0.5)] + 0.25[(0.5)(2.5)]}{4(0.5) + (0.5)(2.5)}$$

$$= 1.33 \text{ in.}$$

*Ans.*

$$\mathbf{F9-11.} \quad \bar{x} = \frac{\Sigma \tilde{x} V}{\Sigma V} = \frac{1[2(7)(6)] + 4[4(2)(3)]}{2(7)(6) + 4(2)(3)}$$

$$= 1.67 \text{ ft}$$

*Ans.*

$$\bar{y} = \frac{\Sigma \tilde{y} V}{\Sigma V} = \frac{3.5[2(7)(6)] + 1[4(2)(3)]}{2(7)(6) + 4(2)(3)}$$

$$= 2.94 \text{ ft}$$

*Ans.*

$$\bar{z} = \frac{\Sigma \tilde{z} V}{\Sigma V} = \frac{3[2(7)(6)] + 1.5[4(2)(3)]}{2(7)(6) + 4(2)(3)}$$

$$= 2.67 \text{ ft}$$

*Ans.*

$$\mathbf{F9-12.} \quad \bar{x} = \frac{\Sigma \tilde{x} V}{\Sigma V}$$

$$= \frac{0.25[0.5(2.5)(1.8)] + 0.25 \left[ \frac{1}{2}(1.5)(1.8)(0.5) \right] + (1.0) \left[ \frac{1}{2}(1.5)(1.8)(0.5) \right]}{0.5(2.5)(1.8) + \frac{1}{2}(1.5)(1.8)(0.5) + \frac{1}{2}(1.5)(1.8)(0.5)}$$

$$= 0.391 \text{ m}$$

*Ans.*

$$\bar{y} = \frac{\Sigma \tilde{y} V}{\Sigma V} = \frac{5.00625}{3.6} = 1.39 \text{ m}$$

$$\bar{z} = \frac{\Sigma \tilde{z} V}{\Sigma V} = \frac{2.835}{3.6} = 0.7875 \text{ m}$$

*Ans.*

$$\mathbf{F9-13.} \quad A = 2\pi \sum \tilde{r} L$$

$$= 2\pi [0.75(1.5) + 1.5(2) + 0.75\sqrt{(1.5)^2 + (2)^2}]$$

*Ans.*

$$= 37.7 \text{ m}^2$$

$$V = 2\pi \sum \tilde{r} A$$

$$= 2\pi [0.75(1.5)(2) + 0.5(\frac{1}{2})(1.5)(2)]$$

*Ans.*

$$= 18.8 \text{ m}^3$$

**F9–14.**  $A = 2\pi \Sigma \tilde{r}L$

$$= 2\pi [1.95\sqrt{(0.9)^2 + (1.2)^2} + 2.4(1.5) + 1.95(0.9) + 1.5(2.7)] \\ = 77.5 \text{ m}^2$$

*Ans.*

$V = 2\pi \Sigma \tilde{r}A$

$= 2\pi [1.8(\frac{1}{2})(0.9)(1.2) + 1.95(0.9)(1.5)]$

$= 22.6 \text{ m}^3$

*Ans.*

**F9–15.**  $A = 2\pi \Sigma \tilde{r}L$

$= 2\pi [7.5(15) + 15(18) + 22.5\sqrt{15^2 + 20^2} + 15(30)] \\ = 8765 \text{ in.}^2$

*Ans.*

$V = 2\pi \Sigma \tilde{r}A$

$= 2\pi [7.5(15)(38) + 20(\frac{1}{2})(15)(20)]$

$= 45,710 \text{ in.}^3$

*Ans.*

**F9–16.**  $A = 2\pi \Sigma \tilde{r}L$

$= 2\pi [\frac{2(1.5)}{\pi}(\frac{\pi(1.5)}{2}) + 1.5(2) + 0.75(1.5)] \\ = 40.1 \text{ m}^2$

*Ans.*

$V = 2\pi \Sigma \tilde{r}A$

$= 2\pi [\frac{4(1.5)}{3\pi}(\frac{\pi(1.5^2)}{4}) + 0.75(1.5)(2)]$

$= 21.2 \text{ m}^3$

*Ans.*

**F9–17.**  $w_b = \rho_w g h b = 1000(9.81)(6)(1)$

$= 58.86 \text{ kN/m}$

$F_R = \frac{1}{2}(58.76)(6) = 176.58 \text{ kN} = 177 \text{ kN}$

*Ans.*

**F9–18.**  $w_b = \gamma_w h b = 62.4(4)(4) = 998.4 \text{ lb/ft}$

$F_R = 998.4(3) = 3.00 \text{ kip}$

*Ans.*

**F9–19.**  $w_b = \rho_w g h_B b = 1000(9.81)(2)(1.5)$

$= 29.43 \text{ kN/m}$

$F_R = \frac{1}{2}(29.43)(\sqrt{(1.5)^2 + (2)^2})$

$= 36.8 \text{ kN}$

*Ans.*

**F9–20.**  $w_A = \rho_w g h_A b = 1000(9.81)(3)(2)$

$= 58.86 \text{ kN/m}$

$w_B = \rho_w g h_B b = 1000(9.81)(5)(2)$

$= 98.1 \text{ kN/m}$

$F_R = \frac{1}{2}(58.86 + 98.1)(2) = 157 \text{ kN}$

*Ans.*

**F9–21.**  $w_A = \gamma_w h_A b = 62.4(6)(2) = 748.8 \text{ lb/ft}$

$w_B = \gamma_w h_B b = 62.4(10)(2) = 1248 \text{ lb/ft}$

$F_R = \frac{1}{2}(748.8 + 1248)(\sqrt{(3)^2 + (4)^2})$

$= 4.99 \text{ kip}$

*Ans.*

## Chapter 10

### F10–1.

$I_x = \int_A y^2 dA = \int_0^{1 \text{ m}} y^2 [(1 - y^{3/2}) dy] = 0.111 \text{ m}^4$

*Ans.*

### F10–2.

$I_x = \int_A y^2 dA = \int_0^{1 \text{ m}} y^2 (y^{3/2} dy) = 0.222 \text{ m}^4$

*Ans.*

### F10–3.

$I_y = \int_A x^2 dA = \int_0^{1 \text{ m}} x^2 (x^{2/3}) dx = 0.273 \text{ m}^4$

*Ans.*

### F10–4.

$I_y = \int_A x^2 dA = \int_0^{1 \text{ m}} x^2 [(1 - x^{2/3}) dx] = 0.0606 \text{ m}^4$

*Ans.*

**F10–5.**  $I_x = [\frac{1}{12}(50)(450^3) + 0] + [\frac{1}{12}(300)(50^3) + 0]$

$= 383(10^6) \text{ mm}^4$

$I_y = [\frac{1}{12}(450)(50^3) + 0]$

$+ 2[\frac{1}{12}(50)(150^3) + (150)(50)(100)^2]$

$= 183(10^6) \text{ mm}^4$

*Ans.*

**F10–6.**  $I_x = \frac{1}{12}(360)(200^3) - \frac{1}{12}(300)(140^3)$

$= 171(10^6) \text{ mm}^4$

*Ans.*

$I_y = \frac{1}{12}(200)(360^3) - \frac{1}{12}(140)(300^3)$

$= 463(10^6) \text{ mm}^4$

*Ans.*

**F10–7.**  $I_y = 2[\frac{1}{12}(50)(200^3) + 0]$

$+ [\frac{1}{12}(300)(50^3) + 0]$

$= 69.8(10^6) \text{ mm}^4$

*Ans.*

### F10–8.

$\bar{y} = \frac{\sum \tilde{y} A}{\sum A} = \frac{15(150)(30) + 105(30)(150)}{150(30) + 30(150)} = 60 \text{ mm}$

$\bar{I}_{x'} = \sum (\bar{I} + Ad^2)$

$= [\frac{1}{12}(150)(30)^3 + (150)(30)(60 - 15)^2]$

$+ [\frac{1}{12}(30)(150)^3 + 30(150)(105 - 60)^2]$

$= 27.0(10^6) \text{ mm}^4$

*Ans.*

## Chapter 11

**F11-1.**  $y_G = 0.75 \sin \theta$        $\delta y_G = 0.75 \cos \theta \delta\theta$   
 $x_C = 2(1.5) \cos \theta$        $\delta x_C = -3 \sin \theta \delta\theta$   
 $\delta U = 0;$        $2W\delta y_G + P\delta x_C = 0$   
 $(294.3 \cos \theta - 3P \sin \theta)\delta\theta = 0$   
 $P = 98.1 \cot \theta|_{\theta=60^\circ} = 56.6 \text{ N}$

*Ans.*

**F11-2.**  $x_A = 5 \cos \theta$        $\delta x_A = -5 \sin \theta \delta\theta$   
 $y_G = 2.5 \sin \theta$        $\delta y_G = 2.5 \cos \theta \delta\theta$   
 $\delta U = 0;$        $-P\delta x_A + (-W\delta y_G) = 0$   
 $(5P \sin \theta - 1226.25 \cos \theta)\delta\theta = 0$   
 $P = 245.25 \cot \theta|_{\theta=60^\circ} = 142 \text{ N}$

*Ans.*

**F11-3.**  $x_B = 0.6 \sin \theta$        $\delta x_B = 0.6 \cos \theta \delta\theta$   
 $y_C = 0.6 \cos \theta$        $\delta y_C = -0.6 \sin \theta \delta\theta$   
 $\delta U = 0;$        $-F_{sp}\delta x_B + (-P\delta y_C) = 0$   
 $-9(10^3) \sin \theta (0.6 \cos \theta \delta\theta)$   
 $-2000(-0.6 \sin \theta \delta\theta) = 0$   
 $\sin \theta = 0$        $\theta = 0^\circ$   
 $-5400 \cos \theta + 1200 = 0$   
 $\theta = 77.16^\circ = 77.2^\circ$

*Ans.*

**F11-4.**  $x_B = 0.9 \cos \theta$        $\delta x_B = -0.9 \sin \theta \delta\theta$   
 $x_C = 2(0.9 \cos \theta)$        $\delta x_C = -1.8 \sin \theta \delta\theta$   
 $\delta U = 0;$        $P\delta x_B + (-F_{sp}\delta x_C) = 0$

*Ans.*

$$\begin{aligned} & 6(10^3)(-0.9 \sin \theta \delta\theta) \\ & -36(10^3)(\cos \theta - 0.5)(-1.8 \sin \theta \delta\theta) = 0 \\ & \sin \theta (64800 \cos \theta - 37800) \delta\theta = 0 \\ & \sin \theta = 0 \quad \theta = 0^\circ \end{aligned}$$

$$\begin{aligned} & 64800 \cos \theta - 37800 = 0 \\ & \theta = 54.31^\circ = 54.3^\circ \end{aligned}$$

*Ans.*

**F11-5.**  $y_G = 2.5 \sin \theta$        $\delta y_G = 2.5 \cos \theta \delta\theta$   
 $x_A = 5 \cos \theta$        $\delta x_C = -5 \sin \theta \delta\theta$   
 $\delta U = 0;$        $(-F_{sp}\delta x_A) - W\delta y_G = 0$   
 $(15000 \sin \theta \cos \theta - 7500 \sin \theta$   
 $-1226.25 \cos \theta)\delta\theta = 0$

$$\begin{aligned} & \theta = 56.33^\circ = 56.3^\circ \end{aligned}$$

*Ans.*

$$\text{or } \theta = 9.545^\circ = 9.55^\circ$$

*Ans.*

**F11-6.**  $F_{sp} = 15000(0.6 - 0.6 \cos \theta)$   
 $x_C = 3[0.3 \sin \theta]$        $\delta x_C = 0.9 \cos \theta \delta\theta$   
 $y_B = 2[0.3 \cos \theta]$        $\delta y_B = -0.6 \sin \theta \delta\theta$   
 $\delta U = 0;$        $P\delta x_C + F_{sp}\delta y_B = 0$   
 $(135 \cos \theta - 5400 \sin \theta + 5400 \sin \theta \cos \theta)\delta\theta = 0$   
 $\theta = 20.9^\circ$

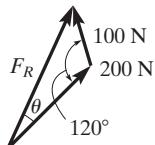
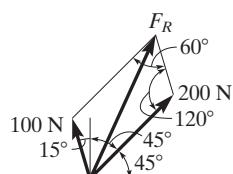
*Ans.*

# Preliminary Problems

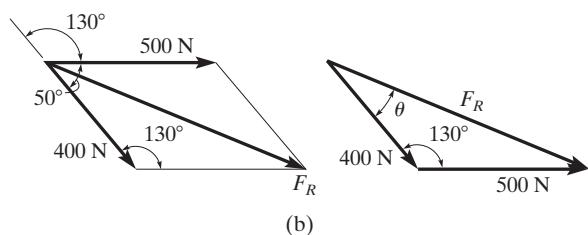
## Statics Solutions

### Chapter 2

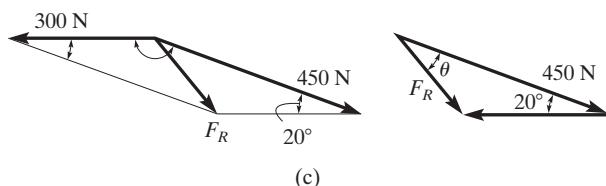
2-1.



(a)

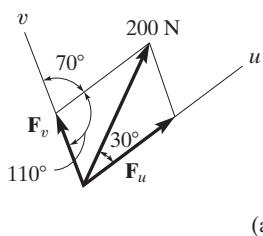


(b)

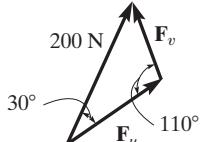


(c)

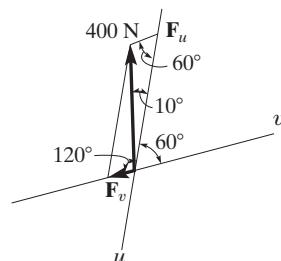
2-2.



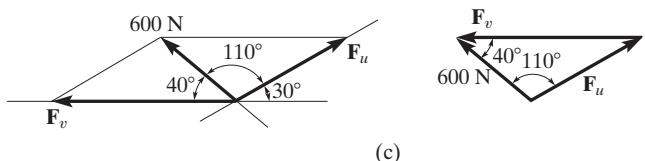
(a)



(b)

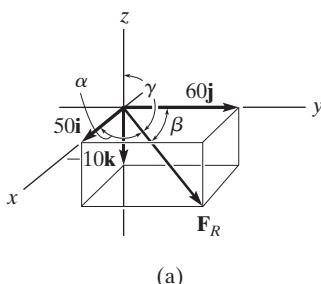


(b)

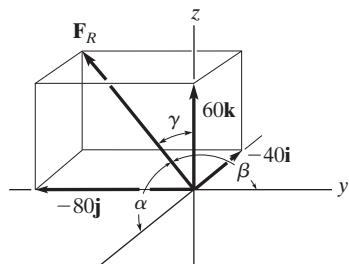


(c)

2-3.



(a)



(b)

2-4. a)  $\mathbf{F} = \{-4\mathbf{i} - 4\mathbf{j} + 2\mathbf{k}\}$  kN

$$F = \sqrt{(4)^2 + (-4)^2 + (2)^2} = 6 \text{ kN}$$

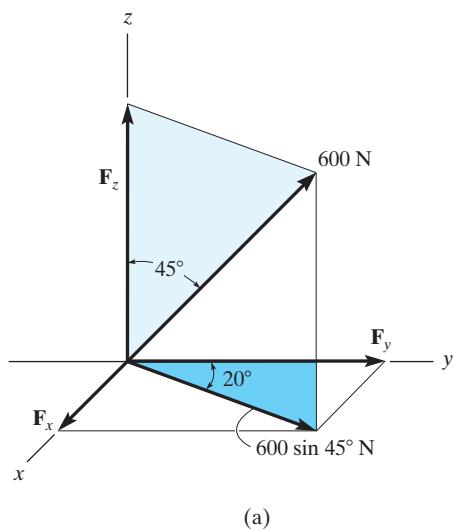
$$\cos \beta = \frac{-2}{3}$$

b)  $\mathbf{F} = \{20\mathbf{i} + 20\mathbf{j} - 10\mathbf{k}\}$  N

$$F = \sqrt{(20)^2 + (20)^2 + (-10)^2} = 30 \text{ N}$$

$$\cos \beta = \frac{2}{3}$$

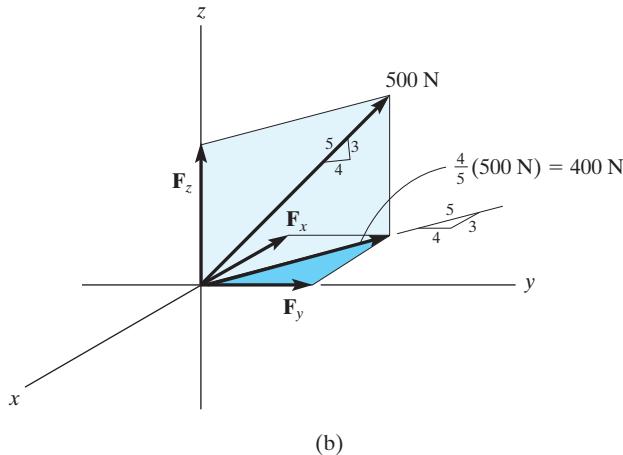
2-5.



$$F_x = (600 \sin 45^\circ) \sin 20^\circ \text{ N}$$

$$F_y = (600 \sin 45^\circ) \cos 20^\circ \text{ N}$$

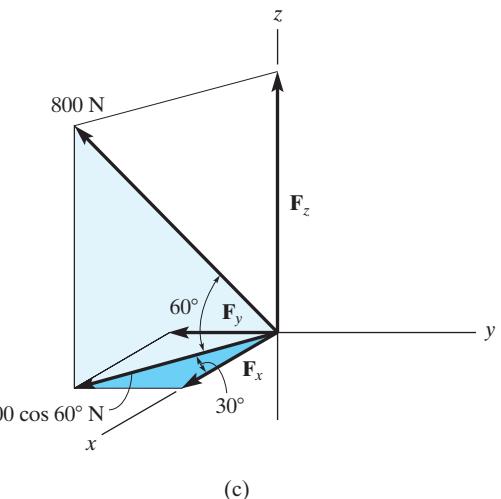
$$F_z = 600 \cos 45^\circ \text{ N}$$



$$F_x = -\frac{3}{5}(400) \text{ N}$$

$$F_y = \frac{4}{5}(400) \text{ N}$$

$$F_z = \frac{3}{5}(500) \text{ N}$$



$$F_x = 800 \cos 60^\circ \cos 30^\circ \text{ N}$$

$$F_y = -800 \cos 60^\circ \sin 30^\circ \text{ N}$$

$$F_z = 800 \sin 60^\circ \text{ N}$$

$$2-6. \quad \text{a) } \mathbf{r}_{AB} = \{-5\mathbf{i} + 3\mathbf{j} - 2\mathbf{k}\} \text{ m}$$

$$\text{b) } \mathbf{r}_{AB} = \{4\mathbf{i} + 8\mathbf{j} - 3\mathbf{k}\} \text{ m}$$

$$\text{c) } \mathbf{r}_{AB} = \{6\mathbf{i} - 3\mathbf{j} - 4\mathbf{k}\} \text{ m}$$

$$2-7. \quad \text{a) } \mathbf{F} = 15 \text{ kN} \left( \frac{-3}{5} \mathbf{i} + \frac{4}{5} \mathbf{j} \right) = \{-9\mathbf{i} + 12\mathbf{j}\} \text{ kN}$$

$$\text{b) } \mathbf{F} = 600 \text{ N} \left( \frac{2}{3} \mathbf{i} + \frac{2}{3} \mathbf{j} - \frac{1}{3} \mathbf{k} \right)$$

$$= \{400\mathbf{i} + 400\mathbf{j} - 200\mathbf{k}\} \text{ N}$$

$$\text{c) } \mathbf{F} = 300 \text{ N} \left( -\frac{2}{3} \mathbf{i} + \frac{2}{3} \mathbf{j} - \frac{1}{3} \mathbf{k} \right)$$

$$= \{-200\mathbf{i} + 200\mathbf{j} - 100\mathbf{k}\} \text{ N}$$

$$2-8. \quad \text{a) } \mathbf{r}_A = \{3\mathbf{k}\} \text{ m}, \quad r_A = 3 \text{ m}$$

$$\mathbf{r}_B = \{2\mathbf{i} + 2\mathbf{j} - 1\mathbf{k}\} \text{ m}, \quad r_B = 3 \text{ m}$$

$$\mathbf{r}_A \cdot \mathbf{r}_B = 0(2) + 0(0) + (3)(-1) = -3 \text{ m}^2$$

$$\mathbf{r}_A \cdot \mathbf{r}_B = r_A r_B \cos \theta$$

$$-3 = 3(3) \cos \theta$$

$$\text{b) } \mathbf{r}_A = \{-2\mathbf{i} + 2\mathbf{j} + 1\mathbf{k}\} \text{ m}, \quad r_A = 3 \text{ m}$$

$$\mathbf{r}_B = \{1.5\mathbf{i} - 2\mathbf{k}\} \text{ m}, \quad r_B = 2.5 \text{ m}$$

$$\mathbf{r}_A \cdot \mathbf{r}_B = (-2)(1.5) + 2(0) + (1)(-2) = -5 \text{ m}^2$$

$$\mathbf{r}_A \cdot \mathbf{r}_B = r_A r_B \cos \theta$$

$$-5 = 3(2.5) \cos \theta$$

**2-9.** a)

$$\mathbf{F} = 300 \text{ N} \left( \frac{2}{3}\mathbf{i} + \frac{2}{3}\mathbf{j} - \frac{1}{3}\mathbf{k} \right) = \{200\mathbf{i} + 200\mathbf{j} - 100\mathbf{k}\} \text{ N}$$

$$\mathbf{u}_a = -\frac{3}{5}\mathbf{i} + \frac{4}{5}\mathbf{j}$$

$$\mathbf{F}_a = \mathbf{F} \cdot \mathbf{u}_a = (200)\left(-\frac{3}{5}\right) + (200)\left(\frac{4}{5}\right) + (-100)(0)$$

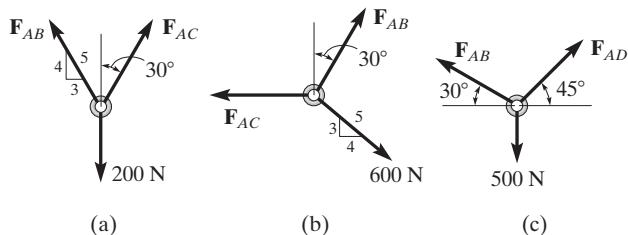
$$\text{b) } \mathbf{F} = 500 \text{ N} \left( -\frac{4}{5}\mathbf{j} + \frac{3}{5}\mathbf{k} \right) = \{-400\mathbf{j} + 300\mathbf{k}\} \text{ N}$$

$$\mathbf{u}_a = -\frac{1}{3}\mathbf{i} + \frac{2}{3}\mathbf{j} + \frac{2}{3}\mathbf{k}$$

$$\mathbf{F}_a = \mathbf{F} \cdot \mathbf{u}_a = (0)\left(-\frac{1}{3}\right) + (-400)\left(\frac{2}{3}\right) + (300)\left(\frac{2}{3}\right)$$

## Chapter 3

**3-1.**



$$\text{3-2. a) } \sum F_x = 0; \quad F \cos 60^\circ - P\left(\frac{1}{\sqrt{2}}\right) - 600\left(\frac{4}{5}\right) = 0$$

$$\sum F_y = 0; \quad -F \sin 60^\circ - P\left(\frac{1}{\sqrt{2}}\right) + 600\left(\frac{3}{5}\right) = 0$$

$$\text{b) } \sum F_x = 0; \quad P\left(\frac{4}{5}\right) - F \sin 60^\circ - 200 \sin 15^\circ = 0$$

$$\sum F_y = 0; \quad -P\left(\frac{3}{5}\right) - F \cos 60^\circ + 200 \cos 15^\circ = 0$$

$$\text{c) } \sum F_x = 0;$$

$$300 \cos 40^\circ + 450 \cos 30^\circ - P \cos 30^\circ + F \sin 10^\circ = 0$$

$$\sum F_y = 0;$$

$$-300 \sin 40^\circ + 450 \sin 30^\circ - P \sin 30^\circ - F \cos 10^\circ = 0$$

## Chapter 4

$$\text{4-1. a) } M_O = 100 \text{ N}(2 \text{ m}) = 200 \text{ N} \cdot \text{m} \curvearrowright$$

$$\text{b) } M_O = -100 \text{ N}(1 \text{ m}) = 100 \text{ N} \cdot \text{m} \curvearrowleft$$

$$\text{c) } M_O = -\left(\frac{3}{5}\right)(500 \text{ N})(2 \text{ m}) = 600 \text{ N} \cdot \text{m} \curvearrowleft$$

$$\text{d) } M_O = \left(\frac{4}{5}\right)(500 \text{ N})(3 \text{ m}) = 1200 \text{ N} \cdot \text{m} \curvearrowright$$

$$\text{e) } M_O = -\left(\frac{3}{5}\right)(100 \text{ N})(5 \text{ m}) = 300 \text{ N} \cdot \text{m} \curvearrowleft$$

$$\text{f) } M_O = 100 \text{ N}(0) = 0$$

$$\text{g) } M_O = -\left(\frac{3}{5}\right)(500 \text{ N})(2 \text{ m}) + \left(\frac{4}{5}\right)(500 \text{ N})(1 \text{ m}) \\ = 200 \text{ N} \cdot \text{m} \curvearrowleft$$

$$\text{h) } M_O = -\left(\frac{3}{5}\right)(500 \text{ N})(3 \text{ m} - 1 \text{ m}) \\ + \left(\frac{4}{5}\right)(500 \text{ N})(1 \text{ m}) = 200 \text{ N} \cdot \text{m} \curvearrowleft$$

$$\text{i) } M_O = \left(\frac{3}{5}\right)(500 \text{ N})(1 \text{ m}) - \left(\frac{4}{5}\right)(500 \text{ N})(3 \text{ m}) \\ = 900 \text{ N} \cdot \text{m} \curvearrowleft$$

$$\text{4-2. } \mathbf{M}_P = \begin{vmatrix} \mathbf{i} & \mathbf{j} & \mathbf{k} \\ 2 & -3 & 0 \\ -3 & 2 & 5 \end{vmatrix} \quad \mathbf{M}_P = \begin{vmatrix} \mathbf{i} & \mathbf{j} & \mathbf{k} \\ 2 & 5 & -1 \\ 2 & -4 & -3 \end{vmatrix}$$

$$\mathbf{M}_P = \begin{vmatrix} \mathbf{i} & \mathbf{j} & \mathbf{k} \\ 5 & -4 & -1 \\ -2 & 3 & 4 \end{vmatrix}$$

$$\text{4-3. a) } M_x = -(100 \text{ N})(3 \text{ m}) = -300 \text{ N} \cdot \text{m}$$

$$M_y = -(200 \text{ N})(2 \text{ m}) = -400 \text{ N} \cdot \text{m}$$

$$M_z = -(300 \text{ N})(2 \text{ m}) = -600 \text{ N} \cdot \text{m}$$

$$\text{b) } M_x = (50 \text{ N})(0.5 \text{ m}) = 25 \text{ N} \cdot \text{m}$$

$$M_y = (400 \text{ N})(0.5 \text{ m}) - (300 \text{ N})(3 \text{ m}) = -700 \text{ N} \cdot \text{m}$$

$$M_z = (100 \text{ N})(3 \text{ m}) = 300 \text{ N} \cdot \text{m}$$

$$\text{c) } M_x = (300 \text{ N})(2 \text{ m}) - (100 \text{ N})(2 \text{ m}) = 400 \text{ N} \cdot \text{m}$$

$$M_y = -(300 \text{ N})(1 \text{ m}) + (50 \text{ N})(1 \text{ m}) \\ + (400 \text{ N})(0.5 \text{ m}) = 250 \text{ N} \cdot \text{m}$$

$$M_z = -(200 \text{ N})(1 \text{ m}) = -200 \text{ N} \cdot \text{m}$$

**4-4. a)**

$$M_a = \begin{vmatrix} -\frac{4}{5} & -\frac{3}{5} & 0 \\ -5 & 2 & 0 \\ 6 & 2 & 3 \end{vmatrix} = \begin{vmatrix} -\frac{4}{5} & -\frac{3}{5} & 0 \\ -1 & 5 & 0 \\ 6 & 2 & 3 \end{vmatrix}$$

b)

$$M_a = \begin{vmatrix} -\frac{1}{\sqrt{2}} & \frac{1}{\sqrt{2}} & 0 \\ 3 & 4 & -2 \\ 2 & -4 & 3 \end{vmatrix} = \begin{vmatrix} -\frac{1}{\sqrt{2}} & \frac{1}{\sqrt{2}} & 0 \\ 5 & 2 & -2 \\ 2 & -4 & 3 \end{vmatrix}$$

$$\text{c) } M_a = \begin{vmatrix} \frac{2}{3} & -\frac{1}{3} & \frac{2}{3} \\ -5 & -4 & 0 \\ 2 & -4 & 3 \end{vmatrix} = \begin{vmatrix} \frac{2}{3} & -\frac{1}{3} & \frac{2}{3} \\ -3 & -5 & 2 \\ 2 & -4 & 3 \end{vmatrix}$$

**4-5.** a)  $\xrightarrow{+}(F_R)_x = \Sigma F_x;$

$$(F_R)_x = -\left(\frac{4}{5}\right)(500 \text{ N}) + 200 \text{ N} = -200 \text{ N}$$

$$+\uparrow(F_R)_y = \Sigma F_y;$$

$$(F_R)_y = -\frac{3}{5}(500 \text{ N}) - 400 \text{ N} = -700 \text{ N}$$

$$\zeta + (M_R)_O = \Sigma M_O;$$

$$(M_R)_O = -\left(\frac{3}{5}\right)(500 \text{ N})(2 \text{ m}) - 400 \text{ N}(4 \text{ m}) \\ = -2200 \text{ N} \cdot \text{m}$$

b)  $\xrightarrow{+}(F_R)_x = \Sigma F_x;$

$$(F_R)_x = \left(\frac{4}{5}\right)(500 \text{ N}) = 400 \text{ N}$$

$$+\uparrow(F_R)_y = \Sigma F_y;$$

$$(F_R)_y = -(300 \text{ N}) - \left(\frac{3}{5}\right)(500 \text{ N}) = -600 \text{ N}$$

$$\zeta + (M_R)_O = \Sigma M_O;$$

$$(M_R)_O = -(300 \text{ N})(2 \text{ m}) - \left(\frac{3}{5}\right)(500 \text{ N})(4 \text{ m}) \\ - 200 \text{ N} \cdot \text{m} = -2000 \text{ N} \cdot \text{m}$$

c)  $\xrightarrow{+}(F_R)_x = \Sigma F_x;$

$$(F_R)_x = \left(\frac{3}{5}\right)(500 \text{ N}) + 100 \text{ N} = 400$$

$$+\uparrow(F_R)_y = \Sigma F_y;$$

$$(F_R)_y = -(500 \text{ N}) - \left(\frac{4}{5}\right)(500 \text{ N}) = -900 \text{ N}$$

$$\zeta + (M_R)_O = \Sigma M_O;$$

$$(M_R)_O = -(500 \text{ N})(2 \text{ m}) - \left(\frac{4}{5}\right)(500 \text{ N})(4 \text{ m}) \\ + \left(\frac{3}{5}\right)(500 \text{ N})(2 \text{ m}) = -2000 \text{ N} \cdot \text{m}$$

d)  $\xrightarrow{+}(F_R)_x = \Sigma F_x;$

$$(F_R)_x = -\left(\frac{4}{5}\right)(500 \text{ N}) + \left(\frac{3}{5}\right)(500 \text{ N}) = -100 \text{ N}$$

$$+\uparrow(F_R)_y = \Sigma F_y;$$

$$(F_R)_y = -\left(\frac{3}{5}\right)(500 \text{ N}) - \left(\frac{4}{5}\right)(500 \text{ N}) = -700 \text{ N}$$

$$\zeta + (M_R)_O = \Sigma M_O;$$

$$(M_R)_O = \left(\frac{4}{5}\right)(500 \text{ N})(4 \text{ m}) + \left(\frac{3}{5}\right)(500 \text{ N})(2 \text{ m}) \\ - \left(\frac{3}{5}\right)(500 \text{ N})(4 \text{ m}) + 200 \text{ N} \cdot \text{m} = 1200 \text{ N} \cdot \text{m}$$

**4-6.** a)  $\xrightarrow{+}(F_R)_x = \Sigma F_x; \quad (F_R)_x = 0$

$$+\uparrow(F_R)_y = \Sigma F_y;$$

$$(F_R)_y = -200 \text{ N} - 260 \text{ N} = -460 \text{ N}$$

$$\zeta + (F_R)_y d = \Sigma M_O;$$

$$-(460 \text{ N})d = -(200 \text{ N})(2 \text{ m}) - (260 \text{ N})(4 \text{ m})$$

$$d = 3.13 \text{ m}$$

Note: Although 460 N acts downward, this is *not* why  $-(460 \text{ N})d$  is negative. It is because the *moment* of 460 N about  $O$  is negative.

b)  $\xrightarrow{+}(F_R)_x = \Sigma F_x;$

$$(F_R)_x = -\left(\frac{3}{5}\right)(500 \text{ N}) = -300 \text{ N}$$

$$+\uparrow(F_R)_y = \Sigma F_y;$$

$$(F_R)_y = -400 \text{ N} - \left(\frac{4}{5}\right)(500 \text{ N}) = -800 \text{ N}$$

$$\zeta + (F_R)_y d = \Sigma M_O;$$

$$-(800 \text{ N})d = -(400 \text{ N})(2 \text{ m}) - \left(\frac{4}{5}\right)(500 \text{ N})(4 \text{ m}) \\ d = 3 \text{ m}$$

c)  $\xrightarrow{+}(F_R)_x = \Sigma F_x;$

$$(F_R)_x = \left(\frac{4}{5}\right)(500 \text{ N}) - \left(\frac{4}{5}\right)(500 \text{ N}) = 0$$

$$+\uparrow(F_R)_y = \Sigma F_y;$$

$$(F_R)_y = -\left(\frac{3}{5}\right)(500 \text{ N}) - \left(\frac{3}{5}\right)(500 \text{ N}) = -600 \text{ N}$$

$$\zeta + (F_R)_y d = \Sigma M_O;$$

$$-(600 \text{ N})d = -\left(\frac{3}{5}\right)(500 \text{ N})(2 \text{ m}) - \left(\frac{3}{5}\right)(500 \text{ N})(4 \text{ m}) \\ - 600 \text{ N} \cdot \text{m} \\ d = 4 \text{ m}$$

**4-7.** a)  $+\downarrow F_R = \Sigma F_z;$

$$F_R = 200 \text{ N} + 100 \text{ N} + 200 \text{ N} = 500 \text{ N}$$

$$(M_R)_x = \Sigma M_x;$$

$$-(500 \text{ N})y = -(100 \text{ N})(2 \text{ m}) - (200 \text{ N})(2 \text{ m}) \\ y = 1.20 \text{ m}$$

$$(M_R)_y = \Sigma M_y;$$

$$(500 \text{ N})x = (100 \text{ N})(2 \text{ m}) + (200 \text{ N})(1 \text{ m})$$

$$x = 0.80 \text{ m}$$

b)  $\downarrow \sum F_R = \Sigma F_z;$

$$F_R = 100 \text{ N} - 100 \text{ N} + 200 \text{ N} = 200 \text{ N}$$

$$(M_R)_x = \Sigma M_x;$$

$$-(200 \text{ N})y = (100 \text{ N})(1 \text{ m}) + (100 \text{ N})(2 \text{ m})$$

$$-(200 \text{ N})(2 \text{ m})$$

$$y = 0.5 \text{ m}$$

$$(M_R)_y = \Sigma M_y;$$

$$(200 \text{ N})x = -(100 \text{ N})(2 \text{ m}) + (100 \text{ N})(2 \text{ m})$$

$$x = 0$$

c)  $\downarrow \sum F_R = \Sigma F_z;$

$$F_R = 400 \text{ N} + 300 \text{ N} + 200 \text{ N} + 100 \text{ N} = 1000 \text{ N}$$

$$(M_R)_x = \Sigma M_x;$$

$$-(1000 \text{ N})y = -(300 \text{ N})(4 \text{ m}) - (100 \text{ N})(4 \text{ m})$$

$$y = 1.6 \text{ m}$$

$$(M_R)_y = \Sigma M_y;$$

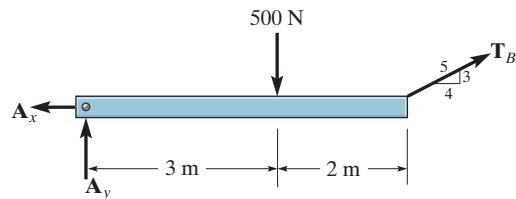
$$(1000 \text{ N})x = (400 \text{ N})(2 \text{ m}) + (300 \text{ N})(2 \text{ m})$$

$$-(200 \text{ N})(2 \text{ m}) - (100 \text{ N})(2 \text{ m})$$

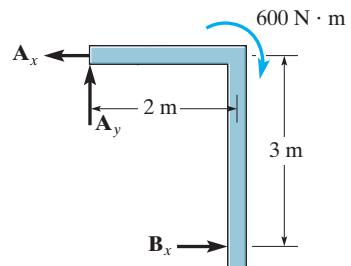
$$x = 0.8 \text{ m}$$

## Chapter 5

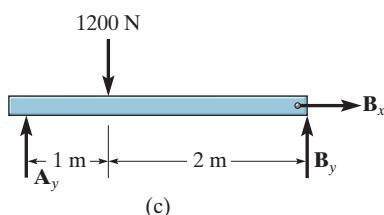
5-1.



(a)

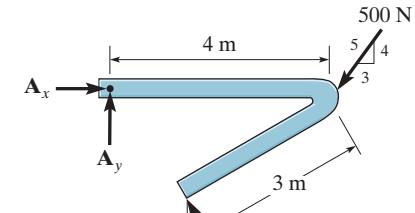


(b)

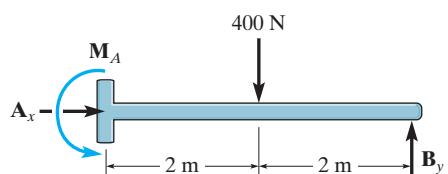


(c)

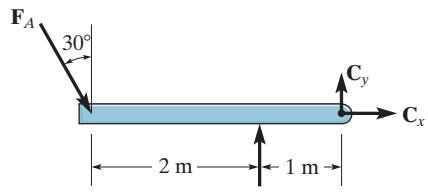
5-2.



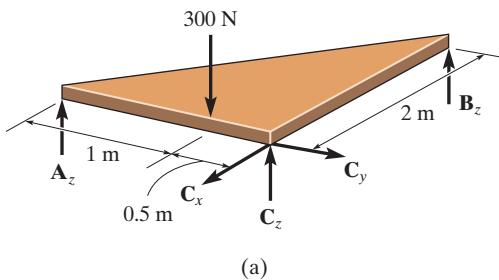
(d)



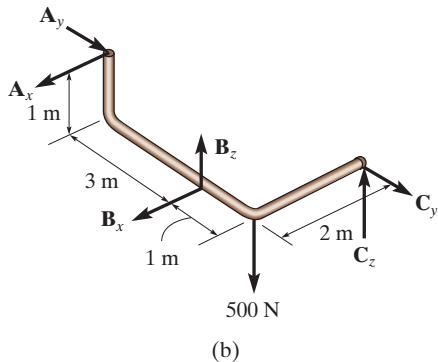
(e)



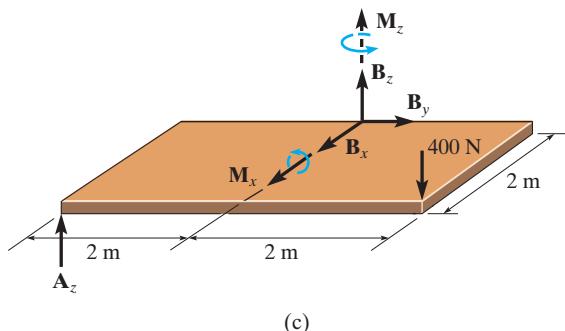
(f)



(a)



(b)



**5-3.** a)  $\sum M_x = 0$ ;

$$-(400 \text{ N})(2 \text{ m}) - (600 \text{ N})(5 \text{ m}) + B_z(5 \text{ m}) = 0$$

$$\Sigma M_y = 0; \quad -A_z(4 \text{ m}) - B_z(4 \text{ m}) = 0$$

$$\Sigma M_z = 0; \quad B_y(4 \text{ m}) - B_x(5 \text{ m}) + (300 \text{ N})(5 \text{ m}) = 0$$

b)  $\sum M_x = 0; \quad A_z(4 \text{ m}) + C_z(6 \text{ m}) = 0$

$$\Sigma M_y = 0; \quad B_z(1 \text{ m}) - C_z(1 \text{ m}) = 0$$

$$\Sigma M_z = 0; \quad -B_y(1 \text{ m}) + (300 \text{ N})(2 \text{ m}) - A_x(4 \text{ m}) + C_y(1 \text{ m}) = 0$$

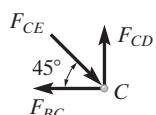
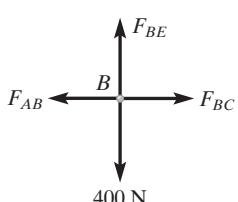
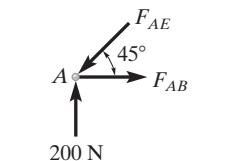
c)  $\sum M_x = 0; \quad B_z(2 \text{ m}) + C_z(3 \text{ m}) - 800 \text{ N} \cdot \text{m} = 0$

$$\Sigma M_y = 0; \quad -C_z(1.5 \text{ m}) = 0$$

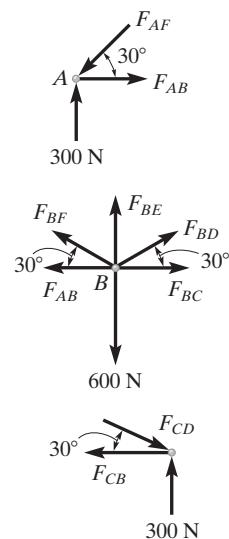
$$\Sigma M_z = 0; \quad -B_x(2 \text{ m}) + C_y(1.5 \text{ m}) = 0$$

## Chapter 6

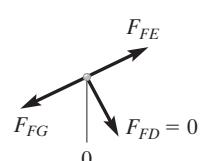
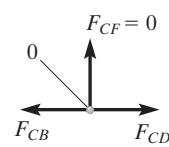
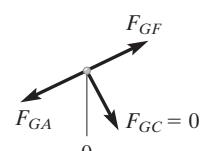
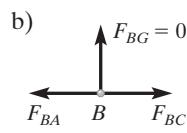
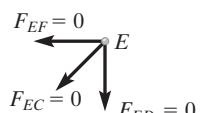
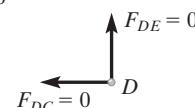
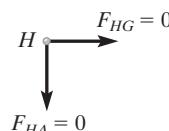
**6-1.** a)  $A_y = 200 \text{ N}, D_x = 0, D_y = 200 \text{ N}$



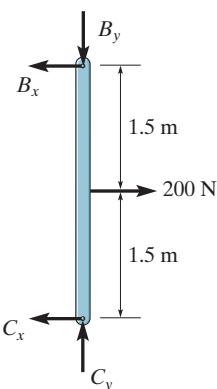
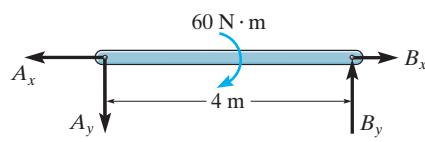
b)  $A_y = 300 \text{ N}, C_x = 0, C_y = 300 \text{ N}$



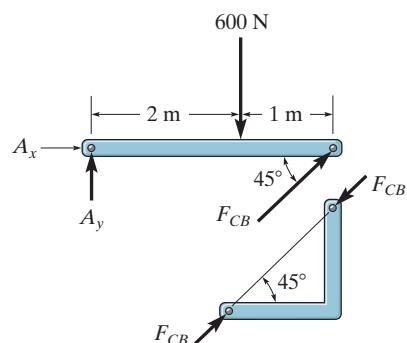
**6-2.** a)



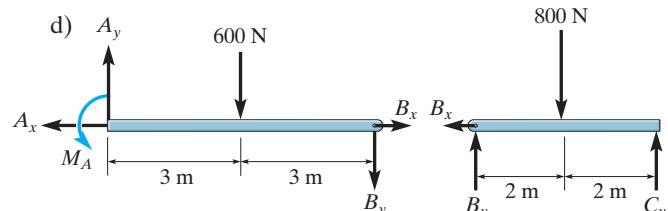
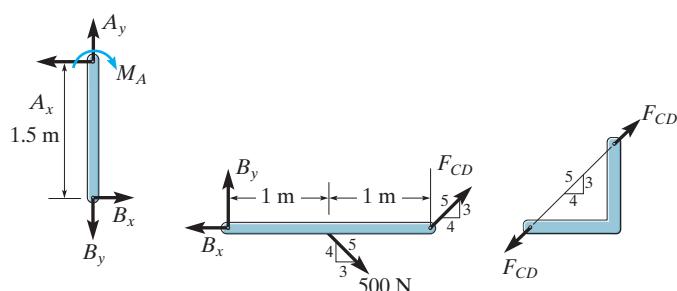
6-3. a)



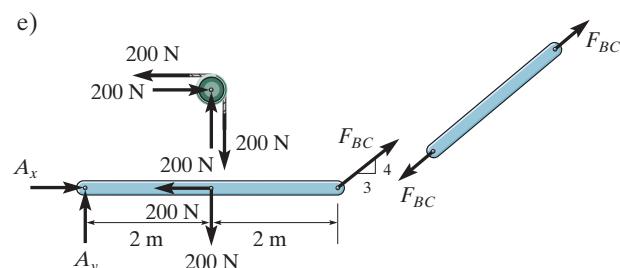
b) CB is a two-force member.



c) CD is a two-force member.

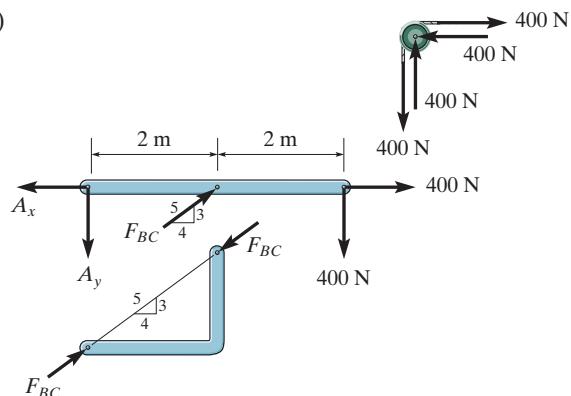


BC is a two-force member.



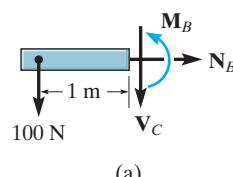
BC is a two-force member.

f)

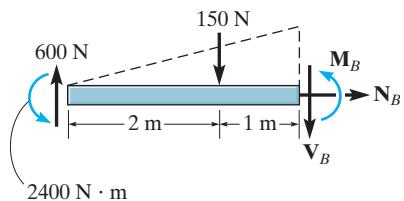


## Chapter 7

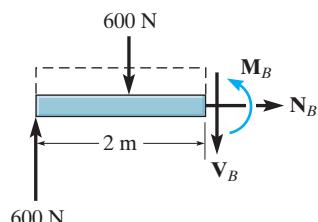
7-1.



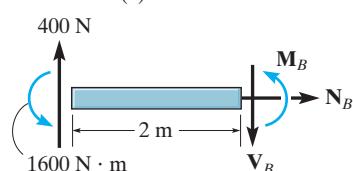
(a)



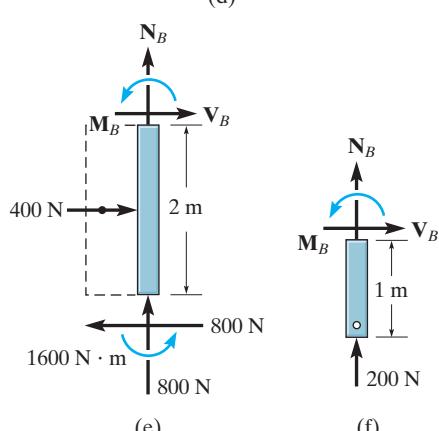
(b)



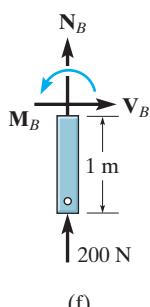
(c)



(d)



(e)

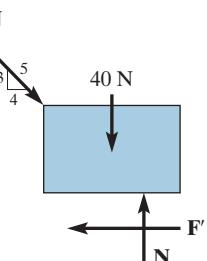


(f)

$$\begin{aligned} \stackrel{+}{\rightarrow} \sum F_x &= 0; \\ \left(\frac{4}{5}\right)(500 \text{ N}) - F' &= 0, F' = 400 \text{ N} \\ +\uparrow \sum F_y &= 0; \\ N - 200 \text{ N} - \left(\frac{3}{5}\right)(500 \text{ N}) &= 0, N = 500 \text{ N} \\ F_{\max} &= 0.3(500 \text{ N}) = 150 \text{ N} < 400 \text{ N} \end{aligned}$$

Slipping  $F = \mu_k N = 0.2(500 \text{ N}) = 100 \text{ N}$  **Ans.**

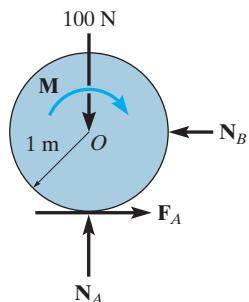
b)



$$\begin{aligned} \stackrel{+}{\rightarrow} \sum F_x &= 0; \\ \frac{4}{5}(100 \text{ N}) - F' &= 0; F' = 80 \text{ N} \\ +\uparrow \sum F_y &= 0; \\ N - 40 \text{ N} - \left(\frac{3}{5}\right)(100 \text{ N}) &= 0; N = 100 \text{ N} \\ F_{\max} &= 0.9(100 \text{ N}) = 90 \text{ N} > 80 \text{ N} \\ F &= F' = 80 \text{ N} \end{aligned}$$

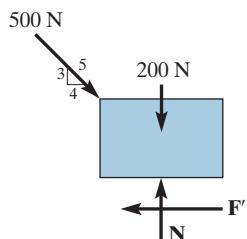
**Ans.**

## 8-2.



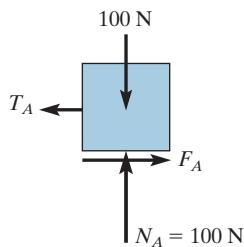
## Chapter 8

8-1. a)



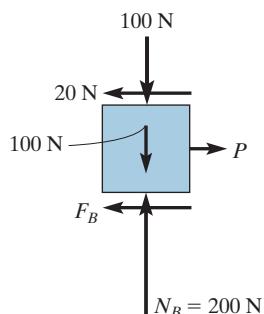
$$\begin{aligned} \text{Require } &F_A = 0.1 N_A \\ +\uparrow \sum F_y &= 0; N_A - 100 \text{ N} = 0 \\ N_A &= 100 \text{ N} \\ F_A &= 0.1(100 \text{ N}) = 10 \text{ N} \\ \zeta + \sum M_O &= 0; -M + (10 \text{ N})(1 \text{ m}) = 0 \\ M &= 10 \text{ N} \cdot \text{m} \end{aligned}$$

- 8-3.** a) Slipping must occur between *A* and *B*.



$$F_A = 0.2(100 \text{ N}) = 20 \text{ N}$$

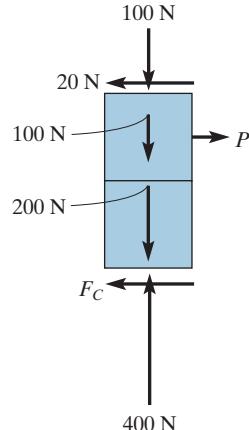
b) Assume *B* slips on *C* and *C* does not slip.



$$F_B = 0.2(200 \text{ N}) = 40 \text{ N}$$

$$\begin{aligned} \pm \sum F_x &= 0; & P - 20 \text{ N} - 40 \text{ N} &= 0 \\ P &= 60 \text{ N} \end{aligned}$$

c) Assume *C* slips and *B* does not slip on *C*.



$$F_C = 0.1(400 \text{ N}) = 40 \text{ N}$$

$$\stackrel{+}{\rightarrow} \sum F_x = 0; \quad P - 20 \text{ N} - 40 \text{ N} = 0$$

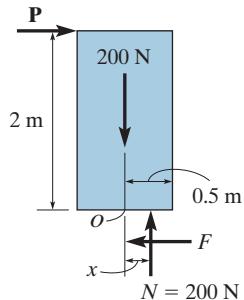
$$P = 60 \text{ N}$$

Therefore,  $P = 60 \text{ N}$

*Ans.*

- 8-4.**

a)



Assume slipping,

$$F = 0.3(200 \text{ N}) = 60 \text{ N}$$

$$\stackrel{+}{\rightarrow} \sum F_x = 0; \quad P - 60 \text{ N} = 0; \quad P = 60 \text{ N}$$

$$\zeta + \sum M_O = 0; \quad 200 \text{ N}(x) - (60 \text{ N})(2 \text{ m}) = 0$$

$$x = 0.6 \text{ m} > 0.5 \text{ m}$$

Block tips,

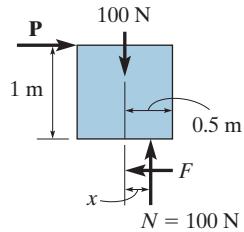
$$x = 0.5 \text{ m}$$

$$\zeta + \sum M_O = 0 \quad (200 \text{ N})(0.5 \text{ m}) - P(2 \text{ m}) = 0$$

$$P = 50 \text{ N}$$

*Ans.*

b)



Assume slipping,

$$F = 0.4(100 \text{ N}) = 40 \text{ N}$$

$$\stackrel{+}{\rightarrow} \sum F_x = 0; \quad P - 40 \text{ N} = 0; \quad P = 40 \text{ N}$$

$$\zeta + \sum M_O = 0; \quad (100 \text{ N})(x) - (40 \text{ N})(1 \text{ m}) = 0$$

$$x = 0.4 \text{ m} < 0.5 \text{ m}$$

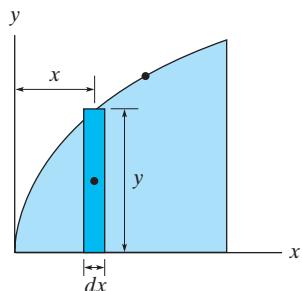
No tipping

$$P = 40 \text{ N}$$

*Ans.*

## Chapter 9

9-1. a)

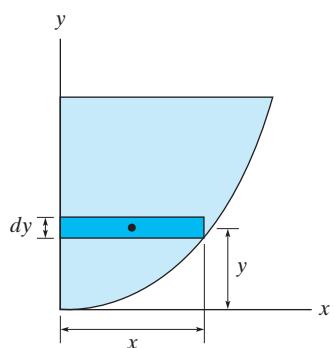


$$\tilde{x} = x$$

$$\tilde{y} = \frac{y}{2} = \frac{\sqrt{x}}{2}$$

$$dA = ydx = \sqrt{x} dx$$

c)

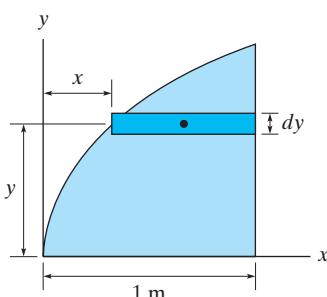


$$\tilde{x} = \frac{x}{2} = \frac{\sqrt{y}}{2}$$

$$\tilde{y} = y$$

$$dA = xdy = \sqrt{y} dy$$

b)

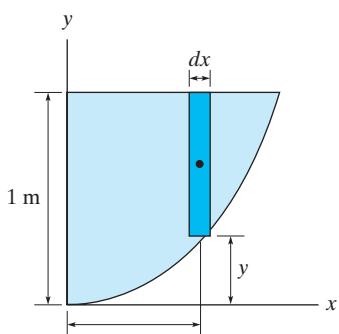


$$\tilde{x} = x + \left(\frac{1-x}{2}\right) = \frac{1+x}{2} = \frac{1+y^2}{2}$$

$$\tilde{y} = y$$

$$dA = (1-x)dy = (1-y^2)dy$$

d)



$$\tilde{x} = x$$

$$\tilde{y} = y + \left(\frac{1-y}{2}\right) = \frac{1+y}{2} = \frac{1+x^2}{2}$$

$$dA = (1-y)dx = (1-x^2)dx$$

# Review Problem Solutions

## Chapter 2

**R2-1.**  $F_R = \sqrt{(300)^2 + (500)^2 - 2(300)(500) \cos 95^\circ} = 605.1 = 605 \text{ N}$

*Ans.*

$$\frac{605.1}{\sin 95^\circ} = \frac{500}{\sin \theta}$$

$$\theta = 55.40^\circ$$

$$\phi = 55.40^\circ + 30^\circ = 85.4^\circ$$

*Ans.*

**R2-2.**  $\frac{F_{1v}}{\sin 30^\circ} = \frac{250}{\sin 105^\circ} \quad F_{1v} = 129 \text{ N}$

*Ans.*

$$\frac{F_{1u}}{\sin 45^\circ} = \frac{250}{\sin 105^\circ} \quad F_{1u} = 183 \text{ N}$$

*Ans.*

**R2-3.**  $F_{Rx} = F_{1x} + F_{2x} + F_{3x} + F_{4x}$   
 $F_{Rx} = -200 + 320 + 180 - 300 = 0$

$$F_{Ry} = F_{1y} + F_{2y} + F_{3y} + F_{4y}$$

$$F_{Ry} = 0 - 240 + 240 + 0 = 0$$

$$\text{Thus, } F_R = 0$$

*Ans.*

**R2-4.**  $\cos^2 30^\circ + \cos^2 70^\circ + \cos^2 \gamma = 1$

$$\cos \gamma = \pm 0.3647$$

$$\gamma = 68.61^\circ \text{ or } 111.39^\circ$$

$$\text{By inspection, } \gamma = 111.39^\circ.$$

$$\mathbf{F} = 250 \{ \cos 30^\circ \mathbf{i} + \cos 70^\circ \mathbf{j} + \cos 111.39^\circ \} \text{ lb}$$

$$= \{ 217\mathbf{i} + 85.5\mathbf{j} - 91.2\mathbf{k} \} \text{ lb}$$

*Ans.*

**R2-5.**  $\mathbf{r} = \{ 50 \sin 20^\circ \mathbf{i} + 50 \cos 20^\circ \mathbf{j} - 35\mathbf{k} \} \text{ ft}$

$$r = \sqrt{(17.10)^2 + (46.98)^2 + (-35)^2} = 61.03 \text{ ft}$$

$$\mathbf{u} = \frac{\mathbf{r}}{r} = (0.280\mathbf{i} + 0.770\mathbf{j} - 0.573\mathbf{k})$$

$$\mathbf{F} = \mathbf{F}\mathbf{u} = \{ 98.1\mathbf{i} + 269\mathbf{j} - 201\mathbf{k} \} \text{ lb}$$

*Ans.*

**R2-6.**  $\mathbf{F}_1 = 600 \left( \frac{4}{5} \right) \cos 30^\circ (+\mathbf{i}) + 600 \left( \frac{4}{5} \right) \sin 30^\circ (-\mathbf{j})$ 

$$+ 600 \left( \frac{3}{5} \right) (+\mathbf{k})$$

$$= \{ 415.69\mathbf{i} - 240\mathbf{j} + 360\mathbf{k} \} \text{ N}$$

*Ans.*

$$\mathbf{F}_2 = 0\mathbf{i} + 450 \cos 45^\circ (+\mathbf{j}) + 450 \sin 45^\circ (+\mathbf{k})$$

$$= \{ 318.20\mathbf{j} + 318.20\mathbf{k} \} \text{ N}$$

*Ans.*

**R2-7.**  $\mathbf{r}_1 = \{ 400\mathbf{i} + 250\mathbf{k} \} \text{ mm}; \quad r_1 = 471.70 \text{ mm}$   
 $\mathbf{r}_2 = \{ 50\mathbf{i} + 300\mathbf{j} \} \text{ mm}; \quad r_2 = 304.14 \text{ mm}$

$$\mathbf{r}_1 \cdot \mathbf{r}_2 = (400)(50) + 0(300) + 250(0) = 20000$$

$$\theta = \cos^{-1} \left( \frac{\mathbf{r}_1 \cdot \mathbf{r}_2}{r_1 r_2} \right) = \cos^{-1} \left( \frac{20000}{(471.70)(304.14)} \right)$$

$$= 82.0^\circ$$

**R2-8.**  $F_{\text{Proj}} = \mathbf{F} \cdot \mathbf{u}_v = (2\mathbf{i} + 4\mathbf{j} + 10\mathbf{k}) \cdot \left( \frac{2}{3}\mathbf{i} + \frac{2}{3}\mathbf{j} - \frac{1}{3}\mathbf{k} \right)$

$$F_{\text{Proj}} = 0.667 \text{ kN}$$

## Chapter 3

**R3-1.**  $\stackrel{+}{\rightarrow} \sum F_x = 0; \quad F_B - F_A \cos 60^\circ - 50 \left( \frac{4}{5} \right) = 0$

$$+ \uparrow \sum F_y = 0; \quad -F_A \sin 60^\circ + 50 \left( \frac{3}{5} \right) = 0$$

$$F_A = 34.6 \text{ lb} \quad F_B = 57.3 \text{ lb} \quad \text{Ans.}$$

**R3-2.**  $\stackrel{+}{\rightarrow} \sum F_x = 0; \quad F_{AC} \cos 30^\circ - F_{AB} = 0 \quad (1)$

$$+ \uparrow \sum F_y = 0; \quad F_{AC} \sin 30^\circ - W = 0 \quad (2)$$

Assuming cable  $AB$  reaches the maximum tension  $F_{AB} = 450 \text{ lb}$ .

From Eq. (1)  $F_{AC} \cos 30^\circ - 450 = 0$

$$F_{AC} = 519.6 \text{ lb} > 480 \text{ lb} \quad (\text{No Good})$$

Assuming cable  $AC$  reaches the maximum tension  $F_{AC} = 480 \text{ lb}$ .

From Eq. (1)  $480 \cos 30^\circ - F_{AB} = 0$

$$F_{AB} = 415.7 \text{ lb} < 450 \text{ lb} \quad (\text{OK})$$

From Eq. (2)  $480 \sin 30^\circ - W = 0 \quad W = 240 \text{ lb}$

*Ans.*

**R3-3.**  $\stackrel{+}{\rightarrow} \sum F_x = 0; \quad F_{AC} \sin 30^\circ - F_{AB} \left( \frac{3}{5} \right) = 0$

$$F_{AC} = 1.20F_{AB} \quad (1)$$

$$+ \uparrow \sum F_y = 0; \quad F_{AC} \cos 30^\circ + F_{AB} \left( \frac{4}{5} \right) - W = 0$$

$$0.8660F_{AC} + 0.8F_{AB} = W \quad (2)$$

Since  $F_{AC} > F_{AB}$ , failure will occur first at cable  $AC$  with  $F_{AC} = 50 \text{ lb}$ . Then solving Eqs. (1) and (2) yields

$$F_{AB} = 41.67 \text{ lb}$$

$$W = 76.6 \text{ lb} \quad \text{Ans.}$$

**R3-4.**  $s_1 = \frac{60}{40} = 1.5 \text{ ft}$

$$+\uparrow \Sigma F_y = 0; \quad F - 2\left(\frac{1}{2}T\right) = 0; \quad F = T$$

$$\pm \Sigma F_x = 0; \quad -F_s + 2\left(\frac{\sqrt{3}}{2}\right)F = 0$$

$$F_s = 1.732F$$

Final stretch is  $1.5 + 0.268 = 1.768 \text{ ft}$

$$40(1.768) = 1.732F$$

$$F = 40.8 \text{ lb} \quad \text{Ans.}$$

**R3-5.**  $\Sigma F_x = 0; \quad -F_1 \sin 45^\circ = 0 \quad F_1 = 0 \quad \text{Ans.}$

$$\Sigma F_z = 0; \quad F_2 \sin 40^\circ - 200 = 0$$

$$F_2 = 311.14 \text{ lb} = 311 \text{ lb} \quad \text{Ans.}$$

Using the results  $F_1 = 0$  and  $F_2 = 311.14 \text{ lb}$  and then summing forces along the  $y$  axis, we have

$$\Sigma F_y = 0; \quad F_3 - 311.14 \cos 40^\circ = 0$$

$$F_3 = 238 \text{ lb} \quad \text{Ans.}$$

**R3-6.**  $\mathbf{F}_1 = \mathbf{F}_1 \{ \cos 60^\circ \mathbf{i} + \sin 60^\circ \mathbf{k} \}$

$$= \{ 0.5F_1 \mathbf{i} + 0.8660F_1 \mathbf{k} \} \text{ N}$$

$$\mathbf{F}_2 = F_2 \left\{ \frac{3}{5} \mathbf{i} - \frac{4}{5} \mathbf{j} \right\}$$

$$= \{ 0.6 F_2 \mathbf{i} - 0.8 F_2 \mathbf{j} \} \text{ N}$$

$$\mathbf{F}_3 = F_3 \{ -\cos 30^\circ \mathbf{i} - \sin 30^\circ \mathbf{j} \}$$

$$= \{ -0.8660F_3 \mathbf{i} - 0.5F_3 \mathbf{j} \} \text{ N}$$

$$\Sigma F_x = 0; \quad 0.5F_1 + 0.6F_2 - 0.8660F_3 = 0$$

$$\Sigma F_y = 0; \quad -0.8F_2 - 0.5F_3 + 800 \sin 30^\circ = 0$$

$$\Sigma F_z = 0; \quad 0.8660F_1 - 800 \cos 30^\circ = 0$$

$$F_1 = 800 \text{ N} \quad F_2 = 147 \text{ N} \quad F_3 = 564 \text{ N} \quad \text{Ans.}$$

**R3-7.**  $\Sigma F_x = 0; \quad F_{CA} \left( \frac{1}{\sqrt{10}} \right) - F_{CB} \left( \frac{1}{\sqrt{10}} \right) = 0$

$$\Sigma F_y = 0; \quad -F_{CA} \left( \frac{3}{\sqrt{10}} \right) - F_{CB} \left( \frac{3}{\sqrt{10}} \right) + F_{CD} \left( \frac{3}{5} \right) = 0$$

$$\Sigma F_z = 0; \quad -500 + F_{CD} \left( \frac{4}{5} \right) = 0$$

Solving:

$$F_{CD} = 625 \text{ lb} \quad F_{CA} = F_{CB} = 198 \text{ lb}$$

**R3-8.**  $\mathbf{F}_{AB} = 700 \left( \frac{2\mathbf{i} + 3\mathbf{j} - 6\mathbf{k}}{\sqrt{2^2 + 3^2 + (-6)^2}} \right)$

$$= \{ 200\mathbf{i} + 300\mathbf{j} - 600\mathbf{k} \} \text{ N}$$

$$\mathbf{F}_{AC} = F_{AC} \left( \frac{-1.5\mathbf{i} + 2\mathbf{j} - 6\mathbf{k}}{\sqrt{(-1.5)^2 + 2^2 + (-6)^2}} \right)$$

$$= -0.2308F_{AC}\mathbf{i} + 0.3077F_{AC}\mathbf{j} - 0.9231F_{AC}\mathbf{k}$$

$$\mathbf{F}_{AD} = F_{AD} \left( \frac{-3\mathbf{i} - 6\mathbf{j} - 6\mathbf{k}}{\sqrt{(-3)^2 + (-6)^2 + (-6)^2}} \right)$$

$$= -0.3333F_{AD}\mathbf{i} - 0.6667F_{AD}\mathbf{j} - 0.6667F_{AD}\mathbf{k}$$

$$\mathbf{F} = \mathbf{Fk}$$

$$\Sigma \mathbf{F} = 0; \quad \mathbf{F}_{AB} + \mathbf{F}_{AC} + \mathbf{F}_{AD} + \mathbf{F} = \mathbf{0}$$

$$(200 - 0.2308F_{AC} - 0.3333F_{AD})\mathbf{i} \\ + (300 + 0.3077F_{AC} - 0.6667F_{AD})\mathbf{j} \\ + (-600 - 0.9231F_{AC} - 0.6667F_{AD} + F)\mathbf{k} = \mathbf{0}$$

$$200 - 0.2308F_{AC} - 0.3333F_{AD} = 0$$

$$300 + 0.3077F_{AC} - 0.6667F_{AD} = 0$$

$$-600 - 0.9231F_{AC} - 0.6667F_{AD} + F = 0$$

$$F_{AC} = 130 \text{ N} \quad F_{AD} = 510 \text{ N}$$

$$F = 1060 \text{ N} = 1.06 \text{ kN} \quad \text{Ans.}$$

## Chapter 4

**R4-1.**  $20(10^3) = 800(16 \cos 30^\circ) + W(30 \cos 30^\circ + 2)$

$$W = 319 \text{ lb} \quad \text{Ans.}$$

**R4-2.**  $\mathbf{F}_R = 50 \text{ lb} \left[ \frac{(10\mathbf{i} + 15\mathbf{j} - 30\mathbf{k})}{\sqrt{(10)^2 + (15)^2 + (-30)^2}} \right]$

$$\mathbf{F}_R = \{ 14.3\mathbf{i} + 21.4\mathbf{j} - 42.9\mathbf{k} \} \text{ lb} \quad \text{Ans.}$$

$$(\mathbf{M}_R)_C = \mathbf{r}_{CB} \times \mathbf{F} = \begin{vmatrix} \mathbf{i} & \mathbf{j} & \mathbf{k} \\ 10 & 45 & 0 \\ 14.29 & 21.43 & -42.86 \end{vmatrix}$$

$$= \{ -1929\mathbf{i} + 428.6\mathbf{j} - 428.6\mathbf{k} \} \text{ lb} \cdot \text{ft} \quad \text{Ans.}$$

**R4-3.**  $\mathbf{r} = \{ 4\mathbf{i} \} \text{ ft}$

$$\mathbf{F} = 24 \text{ lb} \left( \frac{-2\mathbf{i} + 2\mathbf{j} + 4\mathbf{k}}{\sqrt{(-2)^2 + (2)^2 + (4)^2}} \right)$$

$$= \{ -9.80\mathbf{i} + 9.80\mathbf{j} + 19.60\mathbf{k} \} \text{ lb}$$

$$M_y = \begin{vmatrix} 0 & 1 & 0 \\ 4 & 0 & 0 \\ -9.80 & 9.80 & 19.60 \end{vmatrix} = -78.4 \text{ lb} \cdot \text{ft}$$

$$\mathbf{M}_y = \{ -78.4\mathbf{j} \} \text{ lb} \cdot \text{ft} \quad \text{Ans.}$$

**R4-4.**  $(M_c)_R = \Sigma M_z; \quad 0 = 100 - 0.75F$

$$F = 133 \text{ N} \quad \text{Ans.}$$

**R4-5.**  $\pm \sum F_{Rx} = \Sigma F_x; F_{Rx} = 6\left(\frac{5}{13}\right) - 4 \cos 60^\circ = 0.30769 \text{ kN}$

$$+\uparrow \sum F_{Ry} = \Sigma F_y; F_{Ry} = 6\left(\frac{12}{13}\right) - 4 \sin 60^\circ = 2.0744 \text{ kN}$$

$$F_R = \sqrt{(0.30769)^2 + (2.0744)^2} = 2.10 \text{ kN} \quad \text{Ans.}$$

$$\theta = \tan^{-1}\left[\frac{2.0744}{0.30769}\right] = 81.6^\circ \angle \quad \text{Ans.}$$

$$\zeta + M_P = \Sigma M_P; M_P = 8 - 6\left(\frac{12}{13}\right)(7) + 6\left(\frac{5}{13}\right)(5) - 4 \cos 60^\circ(4) + 4 \sin 60^\circ(3)$$

$$M_P = -16.8 \text{ kN} \cdot \text{m} = 16.8 \text{ kN} \cdot \text{m} \quad \text{Ans.}$$

**R4-6.**  $\pm \sum (F_R)_x = \Sigma F_x; (F_R)_x = 200 \cos 45^\circ - 250\left(\frac{4}{5}\right) - 300 = -358.58 \text{ lb} = 358.58 \text{ lb} \leftarrow$

$$+\uparrow (F_R)_y = \Sigma F_y; (F_R)_y = -200 \sin 45^\circ - 250\left(\frac{3}{5}\right) = -291.42 \text{ lb} = 291.42 \text{ lb} \downarrow$$

$$F_R = \sqrt{(F_R)_x^2 + (F_R)_y^2} = \sqrt{358.58^2 + 291.42^2} = 462.07 \text{ lb} = 462 \text{ lb} \quad \text{Ans.}$$

$$\theta = \tan^{-1}\left[\frac{(F_R)_y}{(F_R)_x}\right] = \tan^{-1}\left[\frac{291.42}{358.58}\right] = 39.1^\circ \angle \quad \text{Ans.}$$

$$\zeta + (M_R)_A = \Sigma M_A; 358.58(d) = 250\left(\frac{3}{5}\right)(2.5) + 250\left(\frac{4}{5}\right)(4) + 300(4) - 200 \cos 45^\circ(6) - 200 \sin 45^\circ(3)$$

$$d = 3.07 \text{ ft} \quad \text{Ans.}$$

**R4-7.**  $+\uparrow F_R = \Sigma F_z; F_R = -20 - 50 - 30 - 40 = -140 \text{ kN} = 140 \text{ kN} \downarrow \quad \text{Ans.}$

$$(M_R)_x = \Sigma M_x; -140y = -50(3) - 30(11) - 40(13)$$

$$y = 7.14 \text{ m} \quad \text{Ans.}$$

$$(M_R)_y = \Sigma M_y; 140x = 50(4) + 20(10) + 40(10)$$

$$x = 5.71 \text{ m} \quad \text{Ans.}$$

**R4-8.**  $+\downarrow F_R = \Sigma F; F_R = 12000 + 6000 = 18000 \text{ lb}$

$$F_R = 18.0 \text{ kip} \quad \text{Ans.}$$

$$\zeta + M_{RC} = \Sigma M_C; 18000x = 12000(7.5) + 6000(20)$$

$$x = 11.7 \text{ ft} \quad \text{Ans.}$$

## Chapter 5

**R5-1.**  $\zeta + \Sigma M_A = 0; F(6) + F(4) + F(2) - 3 \cos 45^\circ(2) = 0$

$$F = 0.3536 \text{ kN} = 354 \text{ N} \quad \text{Ans.}$$

**R5-2.**  $\zeta + \Sigma M_A = 0; N_B(7) - 1400(3.5) - 300(6) = 0$

$$N_B = 957.14 \text{ N} = 957 \text{ N} \quad \text{Ans.}$$

$$+\uparrow \Sigma F_y = 0; A_y - 1400 - 300 + 957 = 0 \quad A_y = 743 \text{ N}$$

$$\pm \Sigma F_x = 0; A_x = 0 \quad \text{Ans.}$$

**R5-3.**  $\zeta + \Sigma M_A = 0; 10(0.6 + 1.2 \cos 60^\circ) + 6(0.4)$

$$- N_A(1.2 + 1.2 \cos 60^\circ) = 0$$

$$N_A = 8.00 \text{ kN} \quad \text{Ans.}$$

$$\pm \Sigma F_x = 0; B_x - 6 \cos 30^\circ = 0; B_x = 5.20 \text{ kN} \quad \text{Ans.}$$

$$+\uparrow \Sigma F_y = 0; B_y + 8.00 - 6 \sin 30^\circ - 10 = 0$$

$$B_y = 5.00 \text{ kN} \quad \text{Ans.}$$

**R5-4.**  $\zeta + \Sigma M_A = 0; 50 \cos 30^\circ(20) + 50 \sin 30^\circ(14)$

$$- F_B(18) = 0$$

$$F_B = 67.56 \text{ lb} = 67.6 \text{ lb} \quad \text{Ans.}$$

$$\pm \Sigma F_x = 0; A_x - 50 \sin 30^\circ = 0$$

$$A_x = 25 \text{ lb} \quad \text{Ans.}$$

$$+\uparrow \Sigma F_y = 0; A_y - 50 \cos 30^\circ - 67.56 = 0$$

$$A_y = 110.86 \text{ lb} = 111 \text{ lb} \quad \text{Ans.}$$

**R5-5.**  $\Sigma F_x = 0; A_x = 0 \quad \text{Ans.}$

$$\Sigma F_y = 0; A_y + 200 = 0$$

$$A_y = -200 \text{ N} \quad \text{Ans.}$$

$$\Sigma F_z = 0; A_z - 150 = 0$$

$$A_z = 150 \text{ N} \quad \text{Ans.}$$

$$\Sigma M_x = 0; -150(2) + 200(2) - (M_A)_x = 0$$

$$(M_A)_x = 100 \text{ N} \cdot \text{m} \quad \text{Ans.}$$

$$\Sigma M_y = 0; (M_A)_y = 0 \quad \text{Ans.}$$

$$\Sigma M_z = 0; 200(2.5) - (M_A)_z = 0$$

$$(M_A)_z = 500 \text{ N} \cdot \text{m} \quad \text{Ans.}$$

## R5-6.

$$\Sigma M_y = 0; P(8) - 80(10) = 0 \quad P = 100 \text{ lb} \quad \text{Ans.}$$

$$\Sigma M_x = 0; B_z(28) - 80(14) = 0 \quad B_z = 40 \text{ lb} \quad \text{Ans.}$$

$$\Sigma M_z = 0; -B_x(28) - 100(10) = 0 \quad B_x = -35.7 \text{ lb} \quad \text{Ans.}$$

$$\Sigma F_x = 0; A_x + (-35.7) - 100 = 0 \quad A_x = 136 \text{ lb} \quad \text{Ans.}$$

$$\Sigma F_y = 0; B_y = 0 \quad \text{Ans.}$$

$$\Sigma F_z = 0; A_z + 40 - 80 = 0 \quad A_z = 40 \text{ lb} \quad \text{Ans.}$$

**R5-7.**  $W = (4 \text{ ft})(2 \text{ ft})(2 \text{ lb}/\text{ft}^2) = 16 \text{ lb}$

$$\Sigma F_x = 0; \quad A_x = 0$$

$$\Sigma F_y = 0; \quad A_y = 0$$

$$\Sigma F_z = 0; \quad A_z + B_z + C_z - 16 = 0$$

$$\Sigma M_x = 0; \quad 2B_z - 16(1) + C_z(1) = 0$$

$$\Sigma M_y = 0; \quad -B_z(2) + 16(2) - C_z(4) = 0$$

$$A_z + B_z + C_z = 5.33 \text{ lb}$$

*Ans.*

*Ans.*

*Ans.*

### R5-8.

$$\Sigma F_x = 0; \quad A_x = 0$$

$$\Sigma F_y = 0; \quad 350 - 0.6F_{BC} + 0.6F_{BD} = 0$$

$$\Sigma F_z = 0; \quad A_z - 800 + 0.8F_{BC} + 0.8F_{BD} = 0$$

$$\Sigma M_x = 0; \quad (M_A)_x + 0.8F_{BD}(6) + 0.8F_{BC}(6) - 800(6) = 0$$

$$\Sigma M_y = 0; \quad 800(2) - 0.8F_{BC}(2) - 0.8F_{BD}(2) = 0$$

$$\Sigma M_z = 0; \quad (M_A)_z - 0.6F_{BC}(2) + 0.6F_{BD}(2) = 0$$

$$F_{BD} = 208 \text{ N}$$

$$F_{BC} = 792 \text{ N}$$

$$A_z = 0$$

$$(M_A)_x = 0$$

$$(M_A)_z = 700 \text{ N}\cdot\text{m}$$

## Chapter 6

### R6-1. Joint B:

$$\pm \Sigma F_x = 0; \quad F_{BC} = 3 \text{ kN (C)}$$

$$+\uparrow \Sigma F_y = 0; \quad F_{BA} = 8 \text{ kN (C)}$$

### Joint A:

$$+\uparrow \Sigma F_y = 0; \quad 8.875 - 8 - \frac{3}{5}F_{AC} = 0$$

$$F_{AC} = 1.458 = 1.46 \text{ kN (C)}$$

$$\pm \Sigma F_x = 0; \quad F_{AF} - 3 - \frac{4}{5}(1.458) = 0$$

$$F_{AF} = 4.17 \text{ kN (T)}$$

### Joint C:

$$\pm \Sigma F_x = 0; \quad 3 + \frac{4}{5}(1.458) - F_{CD} = 0$$

$$F_{CD} = 4.167 = 4.17 \text{ kN (C)}$$

$$+\uparrow \Sigma F_y = 0; \quad F_{CF} - 4 + \frac{3}{5}(1.458) = 0$$

$$F_{CF} = 3.125 = 3.12 \text{ kN (C)}$$

### Joint E:

$$\pm \Sigma F_x = 0; \quad F_{EF} = 0$$

$$+\uparrow \Sigma F_y = 0; \quad F_{ED} = 13.125 = 13.1 \text{ kN (C)}$$

### Joint D:

$$+\uparrow \Sigma F_y = 0; \quad 13.125 - 10 - \frac{3}{5}F_{DF} = 0$$

$$F_{DF} = 5.21 \text{ kN (T)} \quad \text{Ans.}$$

### R6-2. Joint A:

$$\pm \Sigma F_x = 0; \quad F_{AB} - F_{AG} \cos 45^\circ = 0$$

$$+\uparrow \Sigma F_y = 0; \quad 333.3 - F_{AG} \sin 45^\circ = 0$$

$$F_{AG} = 471 \text{ lb (C)} \quad \text{Ans.}$$

$$F_{AB} = 333.3 = 333 \text{ lb (T)} \quad \text{Ans.}$$

### Joint B:

$$\pm \Sigma F_x = 0; \quad F_{BC} = 333.3 = 333 \text{ lb (T)} \quad \text{Ans.}$$

$$+\uparrow \Sigma F_y = 0; \quad F_{GB} = 0 \quad \text{Ans.}$$

### Joint D:

$$\pm \Sigma F_x = 0; \quad -F_{DC} + F_{DE} \cos 45^\circ = 0 \quad \text{Ans.}$$

$$+\uparrow \Sigma F_y = 0; \quad 666.7 - F_{DE} \sin 45^\circ = 0$$

$$F_{DE} = 942.9 \text{ lb} = 943 \text{ lb (C)} \quad \text{Ans.}$$

$$F_{DC} = 666.7 \text{ lb} = 667 \text{ lb (T)} \quad \text{Ans.}$$

### Joint E:

$$\pm \Sigma F_x = 0; \quad -942.9 \sin 45^\circ + F_{EG} = 0$$

$$+\uparrow \Sigma F_y = 0; \quad -F_{EC} + 942.9 \cos 45^\circ = 0$$

$$F_{EC} = 666.7 \text{ lb} = 667 \text{ lb (T)} \quad \text{Ans.}$$

$$F_{EG} = 666.7 \text{ lb} = 667 \text{ lb (C)} \quad \text{Ans.}$$

### Joint C:

$$+\uparrow \Sigma F_y = 0; \quad F_{GC} \cos 45^\circ + 666.7 - 1000 = 0$$

$$F_{GC} = 471 \text{ lb (T)} \quad \text{Ans.}$$

$$\mathbf{R6-3.} \quad \zeta + \Sigma M_C = 0; \quad -1000(10) + 1500(20)$$

$$-F_{GJ} \cos 30^\circ(20 \tan 30^\circ) = 0$$

$$F_{GJ} = 2.00 \text{ kip (C)} \quad \text{Ans.}$$

$$+\uparrow \Sigma F_y = 0; \quad -1000 + 2(2000 \cos 60^\circ) - F_{GC} = 0$$

$$F_{GC} = 1.00 \text{ kip (T)} \quad \text{Ans.}$$

### R6-4.

$$+\uparrow \Sigma F_y = 0; \quad 2A_y - 800 - 600 - 800 = 0 \quad A_y = 1100 \text{ lb}$$

$$\pm \Sigma F_x = 0; \quad A_x = 0$$

$$\zeta + \Sigma M_B = 0; \quad F_{GF} \sin 30^\circ(10) + 800(10 - 10 \cos^2 30^\circ)$$

$$-1100(10) = 0$$

$$F_{GF} = 1800 \text{ lb (C)} = 1.80 \text{ kip (C)} \quad \text{Ans.}$$

$$\zeta + \Sigma M_A = 0; \quad F_{FB} \sin 60^\circ(10) - 800(10 \cos^2 30^\circ) = 0$$

$$F_{FB} = 692.82 \text{ lb (T)} = 693 \text{ lb (T)} \quad \text{Ans.}$$

$$\zeta + \Sigma M_F = 0; \quad F_{BC}(15 \tan 30^\circ) + 800(15 - 10 \cos^2 30^\circ)$$

$$-1100(15) = 0$$

$$F_{BC} = 1212.43 \text{ lb (T)} = 1.21 \text{ kip (T)} \quad \text{Ans.}$$

**R6-5.** Joint A:

$$\Sigma F_z = 0; \quad F_{AD} \left( \frac{2}{\sqrt{68}} \right) - 600 = 0$$

$$F_{AD} = 2473.86 \text{ lb (T)} = 2.47 \text{ kip (T)} \quad \text{Ans.}$$

$$\Sigma F_x = 0; \quad F_{AC} \left( \frac{1.5}{\sqrt{66.25}} \right) - F_{AB} \left( \frac{1.5}{\sqrt{66.25}} \right) = 0$$

$$F_{AC} = F_{AB}$$

$$\Sigma F_y = 0; \quad F_{AC} \left( \frac{8}{\sqrt{66.25}} \right) + F_{AB} \left( \frac{8}{\sqrt{66.25}} \right)$$

$$- 2473.86 \left( \frac{8}{\sqrt{68}} \right) = 0$$

$$0.9829 F_{AC} + 0.9829 F_{AB} = 2400$$

$$F_{AC} = F_{AB} = 1220.91 \text{ lb (C)} = 1.22 \text{ kip (C)} \quad \text{Ans.}$$

**R6-6.** CB is a two force member.

Member AC:

$$\zeta + \Sigma M_A = 0; \quad -600(0.75) + 1.5(F_{CB} \sin 75^\circ) = 0$$

$$F_{CB} = 310.6$$

$$B_x = B_y = 310.6 \left( \frac{1}{\sqrt{2}} \right) = 220 \text{ N} \quad \text{Ans.}$$

$$\pm \Sigma F_x = 0; \quad -A_x + 600 \sin 60^\circ - 310.6 \cos 45^\circ = 0$$

$$A_x = 300 \text{ N} \quad \text{Ans.}$$

$$+\uparrow \Sigma F_y = 0; \quad A_y - 600 \cos 60^\circ + 310.6 \sin 45^\circ = 0$$

$$A_y = 80.4 \text{ N} \quad \text{Ans.}$$

**R6-7.** Member AB:

$$\zeta + \Sigma M_A = 0; \quad -750(2) + B_y(3) = 0$$

$$B_y = 500 \text{ N}$$

Member BC:

$$\zeta + \Sigma M_C = 0; \quad -1200(1.5) - 900(1) + B_x(3) - 500(3) = 0$$

$$B_x = 1400 \text{ N}$$

$$+\uparrow \Sigma F_y = 0; \quad A_y - 750 + 500 = 0$$

$$A_y = 250 \text{ N} \quad \text{Ans.}$$

Member AB:

$$\pm \Sigma F_x = 0; \quad -A_x + 1400 = 0$$

$$A_x = 1400 \text{ N} = 1.40 \text{ kN} \quad \text{Ans.}$$

Member BC:

$$\pm \Sigma F_x = 0; \quad C_x + 900 - 1400 = 0$$

$$C_x = 500 \text{ N} \quad \text{Ans.}$$

$$+\uparrow \Sigma F_y = 0; \quad -500 - 1200 + C_y = 0$$

$$C_y = 1700 \text{ N} = 1.70 \text{ kN} \quad \text{Ans.}$$

$$\mathbf{R6-8.} \quad \zeta + \Sigma M_B = 0; \quad F_{CD}(7) - \frac{4}{5}F_{BE}(2) = 0$$

$$\zeta + \Sigma M_A = 0; \quad -150(7)(3.5) + \frac{4}{5}F_{BE}(5) - F_{CD}(7) = 0$$

$$F_{BE} = 1531 \text{ lb} = 1.53 \text{ kip} \quad \text{Ans.}$$

$$F_{CD} = 350 \text{ lb} \quad \text{Ans.}$$

## Chapter 7

$$\mathbf{R7-1.} \quad \zeta + \Sigma M_A = 0; \quad F_{CD}(8) - 150(8 \tan 30^\circ) = 0$$

$$F_{CD} = 86.60 \text{ lb} \quad \text{Ans.}$$

Since member CF is a two-force member,

$$V_D = M_D = 0$$

$$N_D = F_{CD} = 86.6 \text{ lb} \quad \text{Ans.}$$

$$\zeta + \Sigma M_A = 0; \quad B_y(12) - 150(8 \tan 30^\circ) = 0$$

$$B_y = 57.735 \text{ lb} \quad \text{Ans.}$$

$$\pm \Sigma F_x = 0; \quad N_E = 0 \quad \text{Ans.}$$

$$+\uparrow \Sigma F_y = 0; \quad V_E + 57.735 - 86.60 = 0$$

$$V_E = 28.9 \text{ lb} \quad \text{Ans.}$$

$$\zeta + \Sigma M_E = 0; \quad 57.735(9) - 86.60(5) - M_E = 0$$

$$M_E = 86.6 \text{ lb} \cdot \text{ft} \quad \text{Ans.}$$

**R7-2.** Segment DC

$$\pm \Sigma F_x = 0; \quad N_C = 0 \quad \text{Ans.}$$

$$+\uparrow \Sigma F_y = 0; \quad V_C - 3.00 - 6 = 0 \quad V_C = 9.00 \text{ kN} \quad \text{Ans.}$$

$$\zeta + \Sigma M_C = 0; \quad -M_C - 3.00(1.5) - 6(3) - 40 = 0$$

$$M_C = -62.5 \text{ kN} \cdot \text{m} \quad \text{Ans.}$$

Segment DB

$$\pm \Sigma F_x = 0; \quad N_B = 0 \quad \text{Ans.}$$

$$+\uparrow \Sigma F_y = 0; \quad V_B - 10.0 - 7.5 - 4.00 - 6 = 0$$

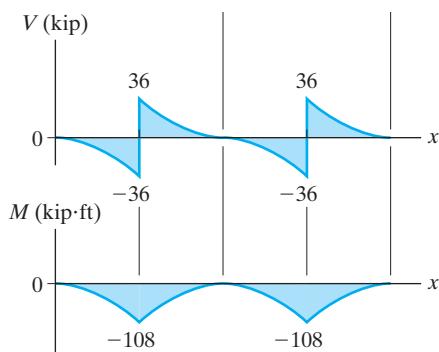
$$V_B = 27.5 \text{ kN} \quad \text{Ans.}$$

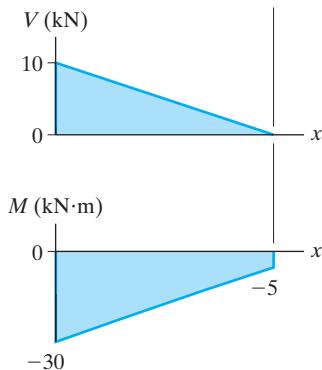
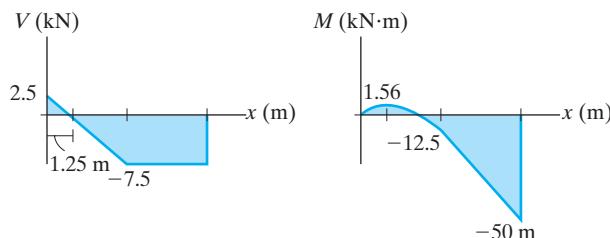
$$\zeta + \Sigma M_B = 0; \quad -M_B - 10.0(2.5) - 7.5(5)$$

$$-4.00(7) - 6(9) - 40 = 0$$

$$M_B = -184.5 \text{ kN} \cdot \text{m} \quad \text{Ans.}$$

**R7-3.**



**R7-4.****R7-5.****R7-6.**

$$\text{At } x = 30 \text{ ft; } y = 3 \text{ ft; } 3 = \frac{F_H}{0.5} \left[ \cosh\left(\frac{0.5}{F_H}(30)\right) - 1 \right]$$

$$F_H = 75.25 \text{ lb}$$

$$\tan \theta_{\max} = \frac{dy}{dx} \Big|_{x=30 \text{ ft}} = \sinh\left(\frac{0.5(30)}{75.25}\right) \quad \theta_{\max} = 11.346^\circ$$

$$T_{\max} = \frac{F_H}{\cos \theta_{\max}} = \frac{75.25}{\cos 11.346^\circ} = 76.7 \text{ lb} \quad \text{Ans.}$$

## Chapter 8

**R8-1.** Assume that the ladder slips at  $A$ :

$$F_A = 0.4 N_A$$

$$+\uparrow \Sigma F_y = 0; \quad N_A - 20 = 0$$

$$N_A = 20 \text{ lb}$$

$$F_A = 0.4(20) = 8 \text{ lb}$$

$$\zeta + \Sigma M_B = 0; \quad P(4) - 20(3) + 20(6) - 8(8) = 0$$

$$P = 1 \text{ lb}$$

*Ans.*

$$+\rightarrow \Sigma F_x = 0; \quad N_B + 1 - 8 = 0$$

$$N_B = 7 \text{ lb} > 0$$

**OK**

The ladder will remain in contact with the wall.

**R8-2.** Crate

$$+\uparrow \Sigma F_y = 0; \quad N_d - 588.6 = 0 \quad N_d = 588.6 \text{ N}$$

$$\pm \Sigma F_x = 0; \quad P - F_d = 0 \quad \text{(1)}$$

$$\zeta + \Sigma M_A = 0; \quad 588.6(x) - P(0.8) = 0 \quad \text{(2)}$$

Crate and dolly

$$+\uparrow \Sigma F_y = 0; \quad N_B + N_A - 588.6 - 98.1 = 0 \quad \text{(3)}$$

$$\pm \Sigma F_x = 0; \quad P - F_A = 0 \quad \text{(4)}$$

$$\zeta + \Sigma M_B = 0; \quad N_A(1.5) - P(1.05)$$

$$- 588.6(0.95) - 98.1(0.75) = 0 \quad \text{(5)}$$

**Friction:** Assuming the crate slips on dolly, then  $F_d = \mu_{sd}N_d = 0.5(588.6) = 294.3 \text{ N}$ . Solving Eqs. (1) and (2)

$$P = 294.3 \text{ N} \quad x = 0.400 \text{ m}$$

Since  $x > 0.3 \text{ m}$ , the crate tips on the dolly. If this is the case  $x = 0.3 \text{ m}$ . Solving Eqs. (1) and (2) with  $x = 0.3 \text{ m}$  yields

$$P = 220.725 \text{ N}$$

$$F_d = 220.725 \text{ N}$$

Assuming the dolly slips at  $A$ , then  $F_A = \mu_{sp}N_A = 0.35N_A$ . Substituting this value into Eqs. (3), (4), and (5) and solving, we have

$$N_A = 559 \text{ N} \quad N_B = 128 \text{ N}$$

$$P = 195.6 \text{ N} = 196 \text{ N} \quad (\text{Controls}) \quad \text{Ans.}$$

**R8-3.** Bar

$$\zeta + \Sigma M_B = 0; \quad P(600) - A_y(900) = 0 \quad A_y = 0.6667P$$

Disk

$$+\uparrow \Sigma F_y = 0; \quad N_C \sin 60^\circ - F_C \sin 30^\circ \\ - 0.6667P - 343.35 = 0 \quad \text{(1)}$$

$$\zeta + \Sigma M_O = 0; \quad F_C(200) - 0.6667P(200) = 0 \quad \text{(2)}$$

**Friction:** If the disk is on the verge of moving, slipping would have to occur at point  $C$ . Hence,  $F_C = \mu_s N_C = 0.2N_C$ . Substituting this into Eqs. (1) and (2) and solving, we have

$$P = 182 \text{ N} \quad \text{Ans.}$$

$$N_C = 606.60 \text{ N}$$

**R8-4.** Cam:

$$\zeta + \Sigma M_O = 0; \quad 5 - 0.4 N_B(0.06) - 0.01(N_B) = 0$$

$$N_B = 147.06 \text{ N}$$

Follower:

$$+\uparrow \Sigma F_y = 0; \quad 147.06 - P = 0$$

$$P = 147 \text{ N} \quad \text{Ans.}$$

$$\mathbf{R8-5.} \quad \dot{\sum} \Sigma F_x = 0; \quad -P + 0.5(1250) = 0$$

$$P = 625 \text{ lb}$$

Assume block  $B$  slips up and block  $A$  does not move.

Block  $A$ :

$$\dot{\sum} \Sigma F_x = 0; \quad F_A - N'' = 0$$

$$+\uparrow \Sigma F_y = 0; \quad N_A - 600 - 0.3N'' = 0$$

Block  $B$ :

$$\dot{\sum} \Sigma F_x = 0; \quad N'' - N' \cos 45^\circ - 0.3N' \sin 45^\circ = 0$$

$$+\uparrow \Sigma F_y = 0; \quad N' \sin 45^\circ - 0.3N' \cos 45^\circ - 150 - 0.3N'' = 0$$

Block  $C$ :

$$\dot{\sum} \Sigma F_x = 0; \quad 0.3N' \cos 45^\circ - N' \cos 45^\circ - 0.5N_C - P = 0$$

$$+\uparrow \Sigma F_y = 0; \quad N_C - N' \sin 45^\circ - 0.3N' \sin 45^\circ - 500 = 0$$

Solving

$$N'' = 629.0 \text{ lb}, N' = 684.3 \text{ lb}, N_C = 838.7 \text{ lb}, P = 1048 \text{ lb},$$

$$N_A = 411.3 \text{ lb}$$

$$F_A = 629.0 \text{ lb} > 0.5(411.3) = 205.6 \text{ lb} \quad \text{No good}$$

All blocks slip at the same time:  $P = 625 \text{ lb}$  Ans.

$$\mathbf{R8-6.} \quad \alpha = \tan^{-1}\left(\frac{10}{25}\right) = 21.80^\circ$$

$$\zeta + \sum M_A = 0; \quad -6000(35) + F_{BD} \cos 21.80^\circ(10) + F_{BD} \sin 21.80^\circ(20) = 0$$

$$F_{BD} = 12\,565 \text{ lb}$$

$$\phi_s = \tan^{-1}(0.4) = 21.80^\circ$$

$$\theta = \tan^{-1}\left(\frac{0.2}{2\pi(0.25)}\right) = 7.256^\circ$$

$$M = Wr \tan(\theta + \phi)$$

$$M = 12\,565(0.25) \tan(7.256^\circ + 21.80^\circ)$$

$$M = 1745 \text{ lb} \cdot \text{in} = 145 \text{ lb} \cdot \text{ft} \quad \text{Ans.}$$

**R8-7.** Block:

$$+\uparrow \Sigma F_y = 0; \quad N - 100 = 0$$

$$N = 100 \text{ lb}$$

$$\dot{\sum} \Sigma F_x = 0; \quad T_1 = 0.4(100) = 0$$

$$T_1 = 40 \text{ lb}$$

$$T_2 = T_1 e^{\mu^\beta}; \quad T_2 = 40 e^{0.4(\frac{\pi}{2})} = 74.978 \text{ lb}$$

System:

$$\zeta + \sum M_A = 0; \quad -100(d) - 40(1) - 50(5) + 74.978(10) = 0$$

$$d = 4.60 \text{ ft}$$

$$\text{Ans.}$$

$$\mathbf{R8-8.} \quad P \approx \frac{Wa}{r}$$

$$= 500(9.81)\left(\frac{2}{40}\right)$$

$$P = 245 \text{ N}$$

## Chapter 9

**R9-1.** Using an element of thickness  $dx$ ,

$$\bar{x} = \frac{\int_A \tilde{x} dA}{\int_A dA} = \frac{\int_a^b x \left(\frac{c^2}{x} dx\right)}{\int_a^b dA} = \frac{\int_a^b c^2 dx}{c^2 \ln \frac{b}{a}} = \frac{c^2 x \Big|_a^b}{c^2 \ln \frac{b}{a}} = \frac{b - a}{\ln \frac{b}{a}}$$

Ans.

**R9-2.** Using an element of thickness  $dx$ ,

$$\begin{aligned} \bar{y} &= \frac{\int_A y dA}{\int_A dA} = \frac{\int_a^b \left(\frac{c^2}{2x}\right) \left(\frac{c^2}{x} dx\right)}{c^2 \ln \frac{b}{a}} = \frac{\int_a^b \frac{c^4}{2x^2} dx}{c^2 \ln \frac{b}{a}} \\ &= \frac{-\frac{c^4}{2x} \Big|_a^b}{c^2 \ln \frac{b}{a}} = \frac{c^2(b - a)}{2ab \ln \frac{b}{a}} \end{aligned}$$

Ans.

$$\mathbf{R9-3.} \quad \bar{z} = \frac{\int_v \tilde{z} dV}{\int_v dV} = \frac{\int_0^a z [\pi(a^2 - z^2) dz]}{\int_0^a \pi(a^2 - z^2) dz}$$

$$= \frac{\pi \left(\frac{a^2 z^2}{2} - \frac{z^4}{4}\right) \Big|_0^a}{\pi \left(a^2 z - \frac{z^3}{3}\right) \Big|_0^a} = \frac{\frac{3}{8}a}{a}$$

Ans.

$$\mathbf{R9-4.} \quad \sum \tilde{x}L = 0(4) + 2(\pi)(2) = 12.5664 \text{ ft}^2$$

$$\sum \tilde{y}L = 0(4) + \frac{2(2)}{\pi}(\pi)(2) = 8 \text{ ft}^2$$

$$\sum \tilde{z}L = 2(4) + 0(\pi)(2) = 8 \text{ ft}^2$$

$$\sum L = 4 + \pi(2) = 10.2832 \text{ ft}$$

$$\tilde{x} = \frac{\sum \tilde{x}L}{\sum L} = \frac{12.5664}{10.2832} = 1.22 \text{ ft}$$

Ans.

$$\tilde{y} = \frac{\sum \tilde{y}L}{\sum L} = \frac{8}{10.2832} = 0.778 \text{ ft}$$

Ans.

$$\tilde{z} = \frac{\sum \tilde{z}L}{\sum L} = \frac{8}{10.2832} = 0.778 \text{ ft}$$

Ans.

**R9-5.**

Segment	$A (\text{mm}^2)$	$\tilde{y} (\text{mm})$	$\tilde{y}A (\text{mm}^3)$
1	300(25)	112.5	843 750
2	100(50)	50	250 000
$\Sigma$	12 500		1 093 750

Thus,

$$\bar{y} = \frac{\Sigma \tilde{y}A}{\Sigma A} = \frac{1093750}{12500} = 87.5 \text{ mm}$$

**R9-6.**

$$\begin{aligned} A &= \Sigma \theta \tilde{r}L \\ &= 2\pi [ 0.6(0.05) + 2(0.6375)\sqrt{(0.025)^2 + (0.075)^2} \\ &\quad + 0.675(0.1) ] \\ &= 1.25 \text{ m}^2 \end{aligned}$$

*Ans.***R9-7.**

$$\begin{aligned} V &= \Sigma \theta \tilde{r}A \\ &= 2\pi \left[ 2(0.65) \left( \frac{1}{2}(0.025)(0.075) \right) + 0.6375(0.05)(0.075) \right] \\ &= 0.0227 \text{ m}^3 \end{aligned}$$

*Ans.*

$$\begin{aligned} \mathbf{R9-8.} \quad dF &= \int dA = 4z^{\frac{1}{3}}(3)dz \\ F &= 12 \int_0^x z^{\frac{1}{3}} dz = 12 \left[ \frac{3}{4}z^{\frac{4}{3}} \right]_0^x = 144 \text{ lb} \quad \text{Ans.} \\ \int_A z dF &= 12 \int_0^8 z^{\frac{4}{3}} dz = 12 \left[ \frac{3}{7}z^{\frac{7}{3}} \right]_0^8 = 658.29 \text{ lb}\cdot\text{ft} \\ \tilde{z} &= \frac{658.29}{144} = 4.57 \text{ ft} \quad \text{Ans.} \end{aligned}$$

**R9-9.**

$$p_a = 1.0(10^3)(9.81)(9) = 88290 \text{ N/m}^2 = 88.29 \text{ kN/m}^2$$

$$p_b = 1.0(10^3)(9.81)(5) = 49050 \text{ N/m}^2 = 49.05 \text{ kN/m}^2$$

Thus,

$$w_A = 88.29(8) = 706.32 \text{ kN/m}$$

$$w_B = 49.05(8) = 392.40 \text{ kN/m}$$

$$F_{R_1} = 392.4(5) = 1962.0 \text{ kN}$$

$$F_{R_2} = \frac{1}{2}(706.32 - 392.4)(5) = 784.8 \text{ kN}$$

$$\zeta + \Sigma M_B = 0; \quad 1962.0(2.5) + 784.8(3.333) - A_y(3) = 0$$

$$A_y = 2507 \text{ kN} = 2.51 \text{ MN} \quad \text{Ans.}$$

$$\pm \Sigma F_x = 0; \quad 784.8\left(\frac{4}{5}\right) + 1962\left(\frac{4}{5}\right) - B_x = 0$$

$$B_x = 2197 \text{ kN} = 2.20 \text{ MN} \quad \text{Ans.}$$

$$+\uparrow \Sigma F_y = 0; \quad 2507 - 784.8\left(\frac{3}{5}\right) - 1962\left(\frac{3}{5}\right) - B_y = 0$$

$$B_y = 859 \text{ kN} \quad \text{Ans.}$$

**R9-10.**

$$A = \int_A dA = \int_{-2}^a -ydx = \int_{-2}^a 2x^2 dx = \frac{2}{3}x^3 \Big|_{-2}^0 = 5.333 \text{ ft}^2$$

$$w = b \gamma h = 1(62.4)(8) = 499.2 \text{ lb}\cdot\text{ft}$$

$$F_y = 5.333(1)(62.4) = 332.8 \text{ lb}$$

$$F_x = \frac{1}{2}(499.2)(8) = 1997 \text{ lb}$$

$$F_N = \sqrt{(332.8)^2 + (1997)^2} = 2024 \text{ lb} = 2.02 \text{ kip} \quad \text{Ans.}$$

**Chapter 10****R10-1.**

$$\begin{aligned} I_x &= \int_A y^2 dA = \int_0^2 y^2(4-x)dy = \int_0^2 y^2(4 - (32)^{\frac{1}{3}}y^{\frac{1}{3}})dy \\ &= 1.07 \text{ in}^4 \end{aligned}$$

*Ans.***R10-2.**

$$\begin{aligned} I_x &= \int_A y^2 dA = \int_0^1 y^2(2x dy) = \int_0^1 y^2(4(1-y)^{\frac{1}{2}})dy \\ &= 0.610 \text{ ft}^4 \end{aligned}$$

*Ans.***R10-3.**

$$\begin{aligned} I_y &= \int_A x^2 dA = 2 \int_0^2 x^2(y dx) = 2 \int_0^2 x^2(1 - 0.25x^2)dx \\ &= 2.13 \text{ ft}^4 \end{aligned}$$

*Ans.*

$$\begin{aligned} \mathbf{R10-4.} \quad dI_{xy} &= d\bar{I}_{x^2y^2} + dA\bar{x}\bar{y} = 0 + \left( y^{\frac{1}{3}} dy \right) \left( \frac{1}{2}y^{\frac{1}{3}} \right)(y) \\ &= \frac{1}{2}y^{\frac{5}{3}} dy \end{aligned}$$

$$I_{xy} = \int dI_{xy} = \int_0^{1 \text{ m}} \frac{1}{2}y^{\frac{5}{3}} dy = \frac{3}{16}y^{\frac{8}{3}} \Big|_0^{1 \text{ m}} = 0.1875 \text{ m}^4 \quad \text{Ans.}$$

$$\mathbf{R10-5.} \quad \frac{s}{h-y} = \frac{b}{h}, \quad s = \frac{b}{h}(h-y)$$

$$(a) dA = s dy = \left[ \frac{b}{h}(h-y) \right] dy$$

$$I_x = \int y^2 dA = \int_0^h y^2 \left[ \frac{b}{h}(h-y) \right] dy = \frac{bh^3}{12} \quad \text{Ans.}$$

$$(b) I_x = \bar{I}_{x'} + A d^2 \quad \frac{bh^3}{12} = \bar{I}_{x'} + \frac{1}{2}bh \left( \frac{h}{3} \right)^2 \quad I_x = \frac{bh^3}{36} \quad \text{Ans.}$$

**R10-6.**  $dI_{xy} = dI_{x^2y^2} + dA\bar{x}\bar{y}$ 

$$= 0 + \left( y^{\frac{1}{3}} dy \right) \left( \frac{1}{2}y^{\frac{1}{3}} \right)(y)$$

$$= \frac{1}{2}y^{\frac{5}{3}} dy$$

$$I_{xy} = \int dI_{xy} = \int_0^{1\text{ m}} \frac{1}{2} y^{\frac{8}{3}} dy = \frac{3}{16} y^{\frac{8}{3}} \Big|_0^{1\text{ m}} = 0.1875 \text{ m}^4 \quad \text{Ans.}$$

$$\begin{aligned} \mathbf{R10-7.} \quad I_y &= \left[ \frac{1}{12}(d)(d^3) + 0 \right] + 4 \left[ \frac{1}{36}(0.2887d) \left( \frac{d}{2} \right)^3 \right. \\ &\quad \left. + \frac{1}{2}(0.2887d) \left( \frac{d}{2} \right) \left( \frac{d}{6} \right)^2 \right] \\ &= 0.0954d^4 \quad \text{Ans.} \end{aligned}$$

$$\begin{aligned} \mathbf{R10-8.} \quad dI_x &= \frac{1}{2}\rho\pi y^4 dx = \frac{1}{2}\rho\pi \left( \frac{b^4}{a^4}x^4 + \frac{4b^4}{a^3}x^3 + \frac{6b^4}{a^2}x^2 \right. \\ &\quad \left. + \frac{4b^4}{a}x + b^4 \right) dx \\ I_x &= \int dI_x = \frac{1}{2}\rho\pi \int_0^a \left( \frac{b^4}{a^4}x^4 + \frac{4b^4}{a^3}x^3 + \frac{6b^4}{a^2}x^2 \right. \\ &\quad \left. + \frac{4b^4}{a}x + b^4 \right) dx \\ &= \frac{31}{10}\rho\pi ab^4 \\ m &= \int dm = \int_0^a \rho\pi y^2 dx \\ &= \rho\pi \int_0^a \left( \frac{b^2}{a^2}x^2 + \frac{2b^2}{a}x + b^2 \right) dx \\ &= \frac{7}{3}\rho\pi ab^2 \\ I_x &= \frac{93}{70}mb^2 \quad \text{Ans.} \end{aligned}$$

## Chapter 11

$$\begin{aligned} \mathbf{R11-1.} \quad x &= 2L \cos \theta \\ \delta x &= -2L \sin \theta \delta \theta \end{aligned}$$

$$y = L \sin \theta$$

$$\delta y = L \cos \theta \delta \theta$$

$$\delta U = 0; \quad -P\delta y - F\delta x = 0$$

$$-PL \cos \theta \delta \theta - F(-2L \sin \theta) \delta \theta = 0$$

$$-P \cos \theta + 2F \sin \theta = 0$$

$$F = \frac{P}{2 \tan \theta} \quad \text{Ans.}$$

$$\begin{aligned} \mathbf{R11-2.} \quad y_B &= 10 \sin \theta & \delta y_B &= 10 \cos \theta \delta \theta \\ y_D &= 5 \sin \theta & \delta y_D &= 5 \cos \theta \delta \theta \\ x_C &= 2(10 \cos \theta) & \delta x_C &= -20 \sin \theta \delta \theta \end{aligned}$$

$$\begin{aligned} \delta U &= 0; \quad -F_{sp}\delta x_C - 2(2\delta y_D - 20\delta y_B + P\delta x_C) = 0 \\ (20F_{sp})\sin \theta - 20P \sin \theta - 220 \cos \theta &= 0 \end{aligned}$$

However, from the spring formula,

$$F_{sp} = kx = 2[2(10 \cos \theta) - 6] = 40 \cos \theta - 12.$$

Substituting

$$(800 \sin \theta \cos \theta - 240 \sin \theta - 220 \cos \theta - 20P \sin \theta) \delta \theta = 0$$

Since  $\delta \theta \neq 0$ , then

$$\begin{aligned} 800 \sin \theta \cos \theta - 240 \sin \theta - 220 \cos \theta - 20P \sin \theta &= 0 \\ P &= 40 \cos \theta - 11 \cot 45^\circ - 12 \end{aligned}$$

At the equilibrium position,  $\theta = 45^\circ$ . Then

$$P = 40 \cos 45^\circ - 11 \cot 45^\circ - 12 = 5.28 \text{ lb} \quad \text{Ans.}$$

**R11-3.** Using the law of cosines,

$$0.4^2 = x_A^2 + 0.1^2 - 2(x_A)(0.1)\cos \theta$$

Differentiating,

$$\begin{aligned} 0 &= 2x_A \delta x_A - 0.2 \delta x_A \cos \theta + 0.2x_A \sin \theta \delta \theta \\ \delta x_A &= \frac{0.2x_A \sin \theta}{0.2 \cos \theta - 2x_A} \delta \theta \end{aligned}$$

$$\delta U = 0; \quad -F \delta x_A - 50 \delta \theta = 0$$

$$\left( \frac{0.2x_A \sin \theta}{0.2 \cos \theta - 2x_A} F - 50 \right) \delta \theta = 0$$

Since  $\delta \theta \neq 0$ , then

$$\begin{aligned} \frac{0.2x_A \sin \theta}{0.2 \cos \theta - 2x_A} F - 50 &= 0 \\ F &= \frac{50(0.2 \cos \theta - 2x_A)}{0.2x_A \sin \theta} \end{aligned}$$

At the equilibrium position,  $\theta = 60^\circ$ ,

$$0.4^2 = x_A^2 + 0.1^2 - 2(x_A)(0.1) \cos 60^\circ$$

$$x_A = 0.4405 \text{ m}$$

$$F = -\frac{50[0.2 \cos 60^\circ - 2(0.4405)]}{0.2(0.4405) \sin 60^\circ} = 512 \text{ N} \quad \text{Ans.}$$

**R11-4.**  $y = 4 \sin \theta$

$$\delta y = 4 \cos \theta \delta \theta$$

$$F_s = 5(4 - 4 \sin \theta)$$

$$\delta U = 0; \quad -10\delta y + F_s \delta y = 0$$

$$[-10 + 20(1 - \sin \theta)](4 \cos \theta \delta \theta) = 0$$

$$\cos \theta = 0 \quad \text{and} \quad 10 - 20 \sin \theta = 0$$

$$\theta = 90^\circ \quad \theta = 30^\circ \quad \text{Ans.}$$

**R11-5.**  $x_B = 0.1 \sin \theta \quad \delta x_B = 0.1 \cos \theta \delta \theta$

$$x_D = 2(0.7 \sin \theta) - 0.1 \sin \theta = 1.3 \sin \theta \quad \delta x_D = 1.3 \cos \theta \delta \theta$$

$$y_G = 0.35 \cos \theta \quad \delta y_G = -0.35 \sin \theta \delta \theta$$

$$\delta U = 0; \quad 2(-49.05\delta y_G) + F_{sp}(\delta x_B - \delta x_D) = 0 \\ (34.335 \sin \theta - 1.2F_{sp} \cos \theta)\delta\theta = 0$$

However, from the spring formula,

$$F_{sp} = kx = 400[2(0.6 \sin \theta) - 0.3] = 480 \sin \theta - 120.$$

Substituting,

$$(34.335 \sin \theta - 576 \sin \theta \cos \theta + 144 \cos \theta)\delta\theta = 0$$

Since  $\delta\theta \neq 0$ , then

$$34.335 \sin \theta - 576 \sin \theta \cos \theta + 144 \cos \theta = 0$$

$$\theta = 15.5^\circ$$

*Ans.*

$$\text{and } \theta = 85.4^\circ$$

*Ans.*

### R11-6.

$$V_g = mgy = 40(9.81)(0.45 \sin \theta + b) = 176.58 \sin \theta + 392.4 b$$

$$V_e = \frac{1}{2}(1500)(0.45 \cos \theta)^2 = 151.875 \cos^2 \theta$$

$$V = V_g + V_e = 176.58 \sin \theta + 151.875 \cos^2 \theta + 392.4 b$$

$$\frac{dV}{d\theta} = 176.58 \cos \theta - 303.75 \cos \theta \sin \theta = 0$$

$$\cos \theta(176.58 - 303.75 \sin \theta) = 0$$

$$\cos \theta = 0 \quad \theta = 90^\circ$$

*Ans.*

$$\theta = 35.54^\circ = 35.5^\circ$$

*Ans.*

$$\frac{d^2V}{d\theta^2} = -176.58 \sin \theta - 303.75 \cos 2\theta$$

$$\text{At } \theta = 90^\circ, \frac{d^2V}{d\theta^2} \Big|_{\theta=90^\circ} = -176.58 \sin 90^\circ - 303.75 \cos 180^\circ$$

$$= 127.17 > 0$$

$$= 127.17 > 0 \quad \text{Stable}$$

*Ans.*

$$\text{At } \theta = 35.54^\circ, \frac{d^2V}{d\theta^2} \Big|_{\theta=35.54^\circ} = -176.58 \sin 35.54^\circ$$

$$- 303.75 \cos 71.09^\circ$$

$$= -201.10 < 0 \quad \text{Unstable}$$

*Ans.*

$$\mathbf{R11-7.} \quad V = V_e + V_g$$

$$= \frac{1}{2}(24)(2 \cos \theta)^2 + \frac{1}{2}(48)(6 \cos \theta)^2 \\ + 100(3 \sin \theta) \\ = 912 \cos^2 \theta + 300 \sin \theta$$

$$\frac{dV}{d\theta} = -1824 \sin \theta \cos \theta + 300 \cos \theta = 0$$

$$\frac{dV}{d\theta} = -912 \sin 2\theta + 300 \cos \theta = 0$$

$$\theta = 90^\circ \quad \text{or} \quad \theta = 9.467^\circ$$

$$\frac{d^2V}{d\theta^2} = -1824 \cos 2\theta - 300 \sin \theta$$

$$\left. \frac{d^2V}{d\theta^2} \right|_{\theta=90^\circ} = -1824 \cos 180^\circ - 300 \sin 90^\circ \\ = 1524 > 0$$

*Ans.*

$$\left. \frac{d^2V}{d\theta^2} \right|_{\theta=9.467^\circ} = -1824 \cos 18.933^\circ - 300 \sin 9.467^\circ \\ = 1774.7 < 0$$

Thus, the system is in unstable equilibrium at  $\theta = 9.47^\circ$ . *Ans.*

$$\mathbf{R11-8.} \quad V = V_e + V_g$$

$$= \frac{1}{2}kx^2 - Wy$$

$$= \frac{1}{2}(16)(2.5 - 2.5 \sin \theta)^2 - 20(2.5 \cos \theta)$$

$$= 50 \sin^2 \theta - 100 \sin \theta - 50 \cos \theta + 50$$

$$\frac{dV}{d\theta} = 100 \sin \theta \cos \theta - 100 \cos \theta + 50 \sin \theta = 0$$

$$\frac{dV}{d\theta} = 50 \sin 2\theta - 100 \cos \theta + 50 \sin \theta = 0$$

$$\theta = 37.77^\circ = 37.8^\circ$$

$$\frac{d^2V}{d\theta^2} = 100 \cos 2\theta + 100 \sin \theta + 50 \cos \theta$$

$$\left. \frac{d^2V}{d\theta^2} \right|_{\theta=37.77^\circ} = 100 \cos 75.55^\circ + 100 \sin 37.77^\circ + 50 \cos 37.77^\circ \\ = 125.7 > 0$$

Thus, the system is in stable equilibrium at  $\theta = 37.8^\circ$ . *Ans.*

# Answers to Selected Problems

## Chapter 1

- 1–1.** a. 78.5 N  
b. 0.392 mN  
c. 7.46 MN
- 1–2.** a. GN/s  
b. Gg/N  
c. GN/(kg · s)
- 1–3.** a. Gg/s  
b. kN/m  
c. kN/(kg · s)
- 1–5.** a. 45.3 MN  
b. 56.8 km  
c. 5.63  $\mu\text{g}$
- 1–6.** a. 58.3 km  
b. 68.5 s  
c. 2.55 kN  
d. 7.56 mg
- 1–7.** a. 0.431 g  
b. 35.3 kN  
c. 5.32 m
- 1–9.** a. km/s  
b. mm  
c. Gs/kg  
d. mm · N
- 1–10.** a. kN · m  
b. Gg/m  
c.  $\mu\text{N}/\text{s}^2$   
d. GN/s
- 1–11.** a. 8.653 s  
b. 8.368 kN  
c. 893 g
- 1–13.** 2.71 Mg/m<sup>3</sup>
- 1–14.** a.  $44.9(10^{-3}) \text{ N}^2$   
b.  $2.79(10^3) \text{ s}^2$   
c. 23.4 s
- 1–15.** 7.41  $\mu\text{N}$
- 1–17.** 1.00 Mg/m<sup>3</sup>
- 1–18.** a. 0.447 kg · m/N  
b. 0.911 kg · s  
c. 18.8 GN/m
- 1–19.** 1.04 kip
- 1–21.**  $F = 10.0 \text{ nN}$ ,  $W_1 = 78.5 \text{ N}$ ,  $W_2 = 118 \text{ N}$

## Chapter 2

- 2–1.**  $F_R = 497 \text{ N}$ ,  $\phi = 155^\circ$
- 2–2.**  $F = 960 \text{ N}$ ,  $\theta = 45.2^\circ$
- 2–3.**  $F_R = 393 \text{ lb}$ ,  $\phi = 353^\circ$

- 2–5.**  $F_{AB} = 314 \text{ lb}$ ,  $F_{AC} = 256 \text{ lb}$
- 2–6.**  $\phi = 1.22^\circ$
- 2–7.**  $(F_1)_v = 2.93 \text{ kN}$ ,  $(F_1)_u = 2.07 \text{ kN}$
- 2–9.**  $F = 616 \text{ lb}$ ,  $\theta = 46.9^\circ$
- 2–10.**  $F_R = 980 \text{ lb}$ ,  $\phi = 19.4^\circ$
- 2–11.**  $F_R = 10.8 \text{ kN}$ ,  $\phi = 3.16^\circ$
- 2–13.**  $F_a = 30.6 \text{ lb}$ ,  $F_b = 26.9 \text{ lb}$
- 2–14.**  $F = 19.6 \text{ lb}$ ,  $F_b = 26.4 \text{ lb}$
- 2–15.**  $F = 917 \text{ lb}$ ,  $\theta = 31.8^\circ$
- 2–17.**  $F_R = 19.2 \text{ N}$ ,  $\theta = 2.37^\circ$
- 2–18.**  $F_R = 19.2 \text{ N}$ ,  $\theta = 2.37^\circ$
- 2–19.**  $\theta = 53.5^\circ$ ,  $F_{AB} = 621 \text{ lb}$
- 2–21.**  $F_R = 257 \text{ N}$ ,  $\phi = 163^\circ$
- 2–22.**  $F_R = 257 \text{ N}$ ,  $\phi = 163^\circ$
- 2–23.**  $\theta = 75.5^\circ$
- 2–25.**  $\theta = 36.3^\circ$ ,  $\phi = 26.4^\circ$
- 2–26.**  $\theta = 54.3^\circ$ ,  $F_A = 686 \text{ N}$
- 2–27.**  $F_R = 1.23 \text{ kN}$ ,  $\theta = 6.08^\circ$
- 2–29.**  $F_B = 1.61 \text{ kN}$ ,  $\theta = 38.3^\circ$
- 2–30.**  $F_R = 4.01 \text{ kN}$ ,  $\phi = 16.2^\circ$
- 2–31.**  $\theta = 90^\circ$ ,  $F_B = 1 \text{ kN}$ ,  $F_R = 1.73 \text{ kN}$
- 2–33.**  $F_R = 983 \text{ N}$ ,  $\theta = 21.8^\circ$
- 2–34.**  $\mathbf{F}_1 = \{200\mathbf{i} + 346\mathbf{j}\} \text{ N}$ ,  $\mathbf{F}_2 = \{177\mathbf{i} - 177\mathbf{j}\} \text{ N}$
- 2–35.**  $F_R = 413 \text{ N}$ ,  $\theta = 24.2^\circ$
- 2–37.**  $F_R = 1.96 \text{ kN}$ ,  $\theta = 4.12^\circ$
- 2–38.**  $\mathbf{F}_1 = \{30\mathbf{i} + 40\mathbf{j}\} \text{ N}$ ,  $\mathbf{F}_2 = \{-20.7\mathbf{i} - 77.3\mathbf{j}\} \text{ N}$ ,  
 $\mathbf{F}_3 = \{30\mathbf{i}\}$ ,  $F_R = 54.2 \text{ N}$ ,  $\theta = 43.5^\circ$
- 2–39.**  $F_{1x} = 141 \text{ N}$ ,  $F_{1y} = 141 \text{ N}$ ,  $F_{2x} = -130 \text{ N}$ ,  
 $F_{2y} = 75 \text{ N}$
- 2–41.**  $F_R = 12.5 \text{ kN}$ ,  $\theta = 64.1^\circ$
- 2–42.**  $\mathbf{F}_1 = \{680\mathbf{i} - 510\mathbf{j}\} \text{ N}$ ,  $\mathbf{F}_2 = \{-312\mathbf{i} - 541\mathbf{j}\} \text{ N}$ ,  
 $\mathbf{F}_3 = \{-530\mathbf{i} + 530\mathbf{j}\} \text{ N}$
- 2–43.**  $F_R = 546 \text{ N}$ ,  $\theta = 253^\circ$
- 2–45.**  $F_R = \sqrt{F_1^2 + F_2^2 + 2F_1F_2 \cos \phi}$ ,  
 $\theta = \tan^{-1}\left(\frac{F_1 \sin \phi}{F_2 + F_1 \cos \phi}\right)$
- 2–46.**  $\theta = 68.6^\circ$ ,  $F_B = 960 \text{ N}$
- 2–47.**  $F_R = 839 \text{ N}$ ,  $\theta = 14.8^\circ$
- 2–49.**  $F_R = 389 \text{ N}$ ,  $\phi' = 42.7^\circ$
- 2–50.**  $\mathbf{F}_1 = \{9.64\mathbf{i} + 11.5\mathbf{j}\} \text{ kN}$ ,  $\mathbf{F}_2 = \{-24\mathbf{i} + 10\mathbf{j}\} \text{ kN}$ ,  
 $\mathbf{F}_3 = \{31.2\mathbf{i} - 18\mathbf{j}\} \text{ kN}$
- 2–51.**  $F_R = 17.2 \text{ kN}$ ,  $\theta = 11.7^\circ$
- 2–53.**  $\mathbf{F}_1 = \{-15.0\mathbf{i} - 26.0\mathbf{j}\} \text{ kN}$ ,  
 $\mathbf{F}_2 = \{-10.0\mathbf{i} + 24.0\mathbf{j}\} \text{ kN}$
- 2–54.**  $F_R = 25.1 \text{ kN}$ ,  $\theta = 185^\circ$
- 2–55.**  $F = 2.03 \text{ kN}$ ,  $F_R = 7.87 \text{ kN}$
- 2–57.**  $F_R = 380 \text{ N}$ ,  $F_1 = 57.8 \text{ N}$

- 2-58.**  $\theta = 86.0^\circ, F = 1.97 \text{ kN}$
- 2-59.**  $F_R = 11.1 \text{ kN}, \theta = 47.7^\circ$
- 2-61.**  $F_x = 40 \text{ N}, F_y = 40 \text{ N}, F_z = 56.6 \text{ N}$
- 2-62.**  $\alpha = 48.4^\circ, \beta = 124^\circ, \gamma = 60^\circ, F = 8.08 \text{ kN}$
- 2-63.**  $F_R = 114 \text{ lb}, \alpha = 62.1^\circ, \beta = 113^\circ, \gamma = 142^\circ$
- 2-65.**  $\mathbf{F}_1 = \{-106\mathbf{i} + 106\mathbf{j} + 260\mathbf{k}\} \text{ N},$   
 $\mathbf{F}_2 = \{250\mathbf{i} + 354\mathbf{j} - 250\mathbf{k}\} \text{ N},$   
 $\mathbf{F}_R = \{144\mathbf{i} + 460\mathbf{j} + 9.81\mathbf{k}\} \text{ N}, F_R = 482 \text{ N},$   
 $\alpha = 72.6^\circ, \beta = 17.4^\circ, \gamma = 88.8^\circ$
- 2-66.**  $\alpha_1 = 111^\circ, \beta_1 = 69.3^\circ, \gamma_1 = 30.0^\circ$
- 2-67.**  $F_3 = 428 \text{ lb}, \alpha = 88.3^\circ, \beta = 20.6^\circ, \gamma = 69.5^\circ$
- 2-69.**  $F_R = 430 \text{ N}, \alpha = 28.9^\circ, \beta = 67.3^\circ, \gamma = 107^\circ$
- 2-70.**  $F_R = 384 \text{ N}, \cos \alpha = 14.8^\circ, \cos \beta = 88.9^\circ,$   
 $\cos \gamma = 105^\circ$
- 2-71.**  $F_1 = 429 \text{ lb}, \alpha_1 = 62.2^\circ, \beta_1 = 110^\circ, \gamma_1 = 145^\circ$
- 2-73.**  $\mathbf{F}_1 = \{72.0\mathbf{i} + 54.0\mathbf{k}\} \text{ N},$   
 $\mathbf{F}_2 = \{53.0\mathbf{i} + 53.0\mathbf{j} + 130\mathbf{k}\} \text{ N}, \mathbf{F}_3 = \{200\mathbf{k}\}$
- 2-74.**  $F_R = 407 \text{ N}, \alpha = 72.1^\circ, \beta = 82.5^\circ, \gamma = 19.5^\circ$
- 2-75.**  $\mathbf{F}_1 = \{14.0\mathbf{j} - 48.0\mathbf{k}\} \text{ lb},$   
 $\mathbf{F}_2 = \{90\mathbf{i} - 127\mathbf{j} + 90\mathbf{k}\} \text{ lb}$
- 2-77.**  $F_R = 610 \text{ N}, \alpha = 19.4^\circ, \beta = 77.5^\circ, \gamma = 105^\circ$
- 2-78.**  $F_2 = 66.4 \text{ lb}, \alpha = 59.8^\circ, \beta = 107^\circ, \gamma = 144^\circ$
- 2-79.**  $\alpha = 124^\circ, \beta = 71.3^\circ, \gamma = 140^\circ$
- 2-81.**  $F_R = 1.55 \text{ kip}, \alpha = 82.4^\circ, \beta = 37.6^\circ, \gamma = 53.4^\circ$
- 2-82.**  $F_R = 1.60 \text{ kN}, \alpha = 82.6^\circ, \beta = 29.4^\circ, \gamma = 61.7^\circ$
- 2-83.**  $\alpha_3 = 139^\circ,$   
 $\beta_3 = 128^\circ, \gamma_3 = 102^\circ, F_{R1} = 387 \text{ N},$   
 $\beta_3 = 60.7^\circ, \gamma_3 = 64.4^\circ, F_{R2} = 1.41 \text{ kN}$
- 2-85.**  $F = 2.02 \text{ kN}, F_y = 0.523 \text{ kN}$
- 2-86.**  $r_{AB} = 397 \text{ mm}$
- 2-87.**  $\mathbf{F} = \{59.4\mathbf{i} - 88.2\mathbf{j} - 83.2\mathbf{k}\} \text{ lb}, \alpha = 63.9^\circ,$   
 $\beta = 131^\circ, \gamma = 128^\circ$
- 2-89.**  $x = 5.06 \text{ m}, y = 3.61 \text{ m}, z = 6.51 \text{ m}$
- 2-90.**  $z = 6.63 \text{ m}$
- 2-91.**  $x = y = 4.42 \text{ m}$
- 2-93.**  $F_R = 1.17 \text{ kN}, \alpha = 66.9^\circ, \beta = 92.0^\circ, \gamma = 157^\circ$
- 2-94.**  $F_R = 1.17 \text{ kN}, \alpha = 68.0^\circ, \beta = 96.8^\circ, \gamma = 157^\circ$
- 2-95.**  $\mathbf{F}_{BA} = \{-109\mathbf{i} + 131\mathbf{j} + 306\mathbf{k}\} \text{ lb},$   
 $\mathbf{F}_{CA} = \{103\mathbf{i} + 103\mathbf{j} + 479\mathbf{k}\} \text{ lb},$   
 $\mathbf{F}_{DA} = \{-52.1\mathbf{i} - 156\mathbf{j} + 365\mathbf{k}\} \text{ lb}$
- 2-97.**  $F_R = 757 \text{ N}, \alpha = 149^\circ, \beta = 90.0^\circ, \gamma = 59.0^\circ$
- 2-98.**  $F = \{-34.3\mathbf{i} + 22.9\mathbf{j} - 68.6\mathbf{k}\} \text{ lb}$
- 2-99.**  $\mathbf{F} = \{13.4\mathbf{i} + 23.2\mathbf{j} + 53.7\mathbf{k}\} \text{ lb}$
- 2-101.**  $\mathbf{F}_A = \{169\mathbf{i} + 33.8\mathbf{j} - 101\mathbf{k}\} \text{ lb},$   
 $\mathbf{F}_B = \{97.6\mathbf{i} + 97.6\mathbf{j} - 58.6\mathbf{k}\} \text{ lb},$   
 $F_R = 338 \text{ lb}, \alpha = 37.8^\circ,$   
 $\beta = 67.1^\circ, \gamma = 118^\circ$
- 2-102.**  $\mathbf{F}_1 = \{389\mathbf{i} - 64.9\mathbf{j} + 64.9\mathbf{k}\} \text{ lb},$   
 $\mathbf{F}_2 = \{-584\mathbf{i} + 97.3\mathbf{j} - 97.3\mathbf{k}\} \text{ lb}$
- 2-103.**  $F_R = 52.2 \text{ lb}, \alpha = 87.8^\circ, \beta = 63.7^\circ, \gamma = 154^\circ$
- 2-105.**  $F = 105 \text{ lb}$

- 2-106.**  $\mathbf{F} = \{466\mathbf{i} + 339\mathbf{j} - 169\mathbf{k}\} \text{ N}$
- 2-107.**  $\mathbf{F} = \{476\mathbf{i} + 329\mathbf{j} - 159\mathbf{k}\} \text{ N}$
- 2-109.**  $F = 52.1 \text{ lb}$
- 2-110.**  $r_{AB} = 10.0 \text{ ft},$   
 $\mathbf{F} = \{-19.1\mathbf{i} - 14.9\mathbf{j} + 43.7\mathbf{k}\} \text{ lb}, \alpha = 112^\circ,$   
 $\beta = 107^\circ, \gamma = 29.0^\circ$
- 2-111.**  $r_{AB} = 592 \text{ mm}, \mathbf{F} = \{-13.2\mathbf{i} - 17.7\mathbf{j} + 20.3\mathbf{k}\} \text{ N}$
- 2-113.**  $(F_{ED})_{||} = 334 \text{ N}, (F_{ED})_{\perp} = 498 \text{ N}$
- 2-114.**  $\theta = 36.4^\circ$
- 2-115.**  $(F_1)_{AC} = 56.3 \text{ N}$
- 2-117.**  $|\text{Proj } F_{AB}| = 70.5 \text{ N}, |\text{Proj } F_{AC}| = 65.1 \text{ N}$
- 2-118.**  $\theta = 31.0^\circ$
- 2-119.**  $F_1 = 18.3 \text{ lb}, F_2 = 35.6 \text{ lb}$
- 2-121.**  $\theta = 100^\circ$
- 2-122.**  $\theta = 19.2^\circ$
- 2-123.**  $F_{BA} = 187 \text{ N}$
- 2-125.**  $F_u = 246 \text{ N}$
- 2-126.**  $F_{||} = 10.5 \text{ lb}$
- 2-127.**  $\theta = 142^\circ$
- 2-129.**  $F_{||} = 0.182 \text{ kN}$
- 2-130.**  $\theta = 74.4^\circ, \phi = 55.4^\circ$
- 2-131.**  $(F_{BC})_{||} = 28.3 \text{ lb}, (F_{BC})_{\perp} = 68.0 \text{ lb}$
- 2-133.**  $\theta = 132^\circ$
- 2-134.**  $\theta = 23.4^\circ$
- 2-135.**  $[(F_{AB})_{||}] = 63.2 \text{ lb}, [(F_{AB})_{\perp}] = 64.1 \text{ lb}$
- 2-137.**  $F_{OA} = 242 \text{ N}$
- 2-138.**  $\theta = 82.9^\circ$
- 2-139.**  $\text{Proj } \mathbf{F}_{AB} = \{0.229\mathbf{i} - 0.916\mathbf{j} + 1.15\mathbf{k}\} \text{ lb}$

## Chapter 3

- 3-1.**  $F_2 = 9.60 \text{ kN}, F_1 = 1.83 \text{ kN}$
- 3-2.**  $\theta = 4.69^\circ, F_1 = 4.31 \text{ kN}$
- 3-3.**  $\theta = 82.2^\circ, F = 3.96 \text{ kN}$
- 3-5.**  $T = 7.20 \text{ kN}, F = 5.40 \text{ kN}$
- 3-6.**  $T = 7.66 \text{ kN}, \theta = 70.1^\circ$
- 3-7.**  $\theta = 20^\circ, T = 30.5 \text{ lb}$
- 3-9.**  $F = 960 \text{ lb}$
- 3-10.**  $\theta = 40^\circ, T_{AB} = 37.6 \text{ lb}$
- 3-11.**  $\theta = 40^\circ, W = 42.6 \text{ lb}$
- 3-13.**  $F_{CA} = 500(10^3) \text{ lb}, F_{AB} = 433(10^3) \text{ lb},$   
 $F_{AD} = 250(10^3) \text{ lb}$
- 3-14.**  $x_{AD} = 0.4905 \text{ m}, x_{AC} = 0.793 \text{ m}, x_{AB} = 0.467 \text{ m}$
- 3-15.**  $m = 8.56 \text{ kg}$
- 3-17.**  $\frac{1}{k_T} = \frac{1}{k_1} + \frac{1}{k_2}$
- 3-18.**  $k = 176 \text{ N/m}$
- 3-19.**  $l_0 = 2.03 \text{ m}$
- 3-21.**  $l = 2.66 \text{ ft}$
- 3-22.**  $F = 158 \text{ N}$
- 3-23.**  $d = 1.56 \text{ m}$
- 3-25.**  $y = 2 \text{ m}, F_1 = 833 \text{ N}$

- 3–26.**  $T_{HA} = 294 \text{ N}$ ,  $T_{AB} = 340 \text{ N}$ ,  $T_{AE} = 170 \text{ N}$ ,  
 $T_{BD} = 490 \text{ N}$ ,  $T_{BC} = 562 \text{ N}$
- 3–27.**  $m = 26.7 \text{ kg}$
- 3–29.**  $F_{DE} = 392 \text{ N}$ ,  $F_{CD} = 340 \text{ N}$ ,  $F_{CB} = 275 \text{ N}$ ,  
 $F_{CA} = 243 \text{ N}$
- 3–30.**  $m = 20.4 \text{ kg}$
- 3–31.**  $s = 3.38 \text{ m}$ ,  $F = 76.0 \text{ N}$
- 3–33.**  $T_{AB} = 11.0 \text{ lb}$ ,  $T_{AC} = 7.76 \text{ lb}$ ,  $T_{BC} = 11.0 \text{ lb}$ ,  
 $T_{BE} = 19.0 \text{ lb}$ ,  $T_{CD} = 17.4 \text{ lb}$ ,  $\theta = 18.4^\circ$
- 3–34.**  $\theta = 18.4^\circ$ ,  $W = 15.8 \text{ lb}$
- 3–35.**  $F_{AB} = 175 \text{ lb}$ ,  $l = 2.34 \text{ ft}$ , or  
 $F_{AB} = 82.4 \text{ lb}$ ,  $l = 1.40 \text{ ft}$
- 3–37.**  $m_B = 3.58 \text{ kg}$ ,  $N = 19.7 \text{ N}$
- 3–38.**  $F_{AB} = 98.6 \text{ N}$ ,  $F_{AC} = 267 \text{ N}$
- 3–39.**  $d = 2.42 \text{ m}$
- 3–41.**  $T = 30.6 \text{ lb}$ ,  $x = 1.92 \text{ ft}$
- 3–42.**  $W_B = 18.3 \text{ lb}$
- 3–43.**  $F_{AD} = 763 \text{ N}$ ,  $F_{AC} = 392 \text{ N}$ ,  $F_{AB} = 523 \text{ N}$
- 3–45.**  $F_{DA} = 10.0 \text{ lb}$ ,  $F_{DB} = 1.11 \text{ lb}$ ,  $F_{DC} = 15.6 \text{ lb}$
- 3–46.**  $s_{OB} = 327 \text{ mm}$ ,  $s_{OA} = 218 \text{ mm}$
- 3–47.**  $F_{AB} = 219 \text{ N}$ ,  $F_{AC} = F_{AD} = 54.8 \text{ N}$
- 3–49.**  $m = 102 \text{ kg}$
- 3–50.**  $F_{AC} = 113 \text{ lb}$ ,  $F_{AB} = 257 \text{ lb}$ ,  $F_{AD} = 210 \text{ lb}$
- 3–51.**  $F = 1558 \text{ lb}$
- 3–53.**  $F_{AD} = 557 \text{ lb}$ ,  $W = 407 \text{ lb}$
- 3–54.**  $F_{AB} = 79.2 \text{ lb}$ ,  $F_{AC} = 119 \text{ lb}$ ,  $F_{AD} = 283 \text{ lb}$
- 3–55.**  $W_C = 265 \text{ lb}$
- 3–57.**  $W = 55.8 \text{ N}$
- 3–58.**  $F_{AB} = 441 \text{ N}$ ,  $F_{AC} = 515 \text{ N}$ ,  $F_{AD} = 221 \text{ N}$
- 3–59.**  $F_{AB} = 348 \text{ N}$ ,  $F_{AC} = 413 \text{ N}$ ,  $F_{AD} = 174 \text{ N}$
- 3–61.**  $F_{AC} = 85.8 \text{ N}$ ,  $F_{AB} = 578 \text{ N}$ ,  $F_{AD} = 565 \text{ N}$
- 3–62.**  $m = 88.5 \text{ kg}$
- 3–63.**  $F_{AD} = 1.56 \text{ kN}$ ,  $F_{BD} = 521 \text{ N}$ ,  $F_{CD} = 1.28 \text{ kN}$
- 3–65.**  $F_{AE} = 2.91 \text{ kip}$ ,  $F = 1.61 \text{ kip}$
- 3–66.**  $F_{AB} = 360 \text{ lb}$ ,  $F_{AC} = 180 \text{ lb}$ ,  $F_{AD} = 360 \text{ lb}$
- 3–67.**  $W = 375 \text{ lb}$

## Chapter 4

- 4–5.**  $(M_{F_1})_B = 4.125 \text{ kip} \cdot \text{ft} \curvearrowright$ ,  
 $(M_{F_2})_B = 2.00 \text{ kip} \cdot \text{ft} \curvearrowright$ ,  
 $(M_{F_3})_B = 40.0 \text{ lb} \cdot \text{ft} \curvearrowright$
- 4–6.**  $M_P = 341 \text{ in.} \cdot \text{lb} \curvearrowright$   
 $M_F = 403 \text{ in.} \cdot \text{lb} \curvearrowleft$   
 Not sufficient
- 4–7.**  $(M_{F_1})_A = 433 \text{ N} \cdot \text{m} \curvearrowright$   
 $(M_{F_2})_A = 1.30 \text{ kN} \cdot \text{m} \curvearrowright$   
 $(M_{F_3})_A = 800 \text{ N} \cdot \text{m} \curvearrowleft$
- 4–9.**  $M_B = 90.6 \text{ lb} \cdot \text{ft} \curvearrowright$ ,  $M_C = 141 \text{ lb} \cdot \text{ft} \curvearrowleft$
- 4–10.**  $M_A = 195 \text{ lb} \cdot \text{ft} \curvearrowright$
- 4–11.**  $(M_O)_{\max} = 48.0 \text{ kN} \cdot \text{m} \curvearrowright$ ,  $x = 9.81 \text{ m}$
- 4–13.**  $M_B = \{-3.36\mathbf{k}\} \text{ N} \cdot \text{m}$ ,  $\alpha = 90^\circ$ ,  $\beta = 90^\circ$ ,  
 $\gamma = 180^\circ$
- 4–14.**  $\mathbf{M}_O = \{0.5\mathbf{i} + 0.866\mathbf{j} - 3.36\mathbf{k}\} \text{ N} \cdot \text{m}$ ,  
 $\alpha = 81.8^\circ$ ,  $\beta = 75.7^\circ$ ,  $\gamma = 163^\circ$
- 4–15.**  $(M_A)_C = 768 \text{ lb} \cdot \text{ft} \curvearrowright$   
 $(M_A)_B = 636 \text{ lb} \cdot \text{ft} \curvearrowleft$   
 Clockwise
- 4–17.**  $m = \left( \frac{l}{d+l} \right) M$
- 4–18.**  $M_P = (537.5 \cos \theta + 75 \sin \theta) \text{ lb} \cdot \text{ft}$
- 4–19.**  $F = 239 \text{ lb}$
- 4–21.**  $F = 27.6 \text{ lb}$
- 4–22.**  $r = 13.3 \text{ mm}$
- 4–23.**  $(M_R)_A = (M_R)_B = 76.0 \text{ kN} \cdot \text{m} \curvearrowright$
- 4–25.**  $(M_{AB})_A = 3.88 \text{ kip} \cdot \text{ft} \curvearrowright$ ,  
 $(M_{BCD})_A = 2.05 \text{ kip} \cdot \text{ft} \curvearrowright$ ,  
 $(M_{man})_A = 2.10 \text{ kip} \cdot \text{ft} \curvearrowright$
- 4–26.**  $(M_R)_A = 8.04 \text{ kip} \cdot \text{ft} \curvearrowright$
- 4–27.**  $\mathbf{M}_O = \{-40\mathbf{i} - 44\mathbf{j} - 8\mathbf{k}\} \text{ kN} \cdot \text{m}$
- 4–29.**  $\mathbf{M}_O = \{-25\mathbf{i} + 6200\mathbf{j} - 900\mathbf{k}\} \text{ lb} \cdot \text{ft}$
- 4–30.**  $\mathbf{M}_A = \{-175\mathbf{i} + 5600\mathbf{j} - 900\mathbf{k}\} \text{ lb} \cdot \text{ft}$
- 4–31.**  $\mathbf{M}_P = \{-24\mathbf{i} + 24\mathbf{j} + 8\mathbf{k}\} \text{ kN} \cdot \text{m}$
- 4–33.**  $\mathbf{M}_B = \{-110\mathbf{i} - 180\mathbf{j} - 420\mathbf{k}\} \text{ N} \cdot \text{m}$
- 4–34.**  $\mathbf{M}_A = \{574\mathbf{i} + 350\mathbf{j} + 1385\mathbf{k}\} \text{ N} \cdot \text{m}$
- 4–35.**  $F = 585 \text{ N}$
- 4–37.**  $\mathbf{M}_O = \{163\mathbf{i} - 346\mathbf{j} - 360\mathbf{k}\} \text{ N} \cdot \text{m}$
- 4–38.**  $\mathbf{M}_A = \{-82.9\mathbf{i} + 41.5\mathbf{j} + 232\mathbf{k}\} \text{ lb} \cdot \text{ft}$
- 4–39.**  $\mathbf{M}_B = \{-82.9\mathbf{i} - 96.8\mathbf{j} - 52.8\mathbf{k}\} \text{ lb} \cdot \text{ft}$
- 4–41.**  $F = 18.6 \text{ lb}$
- 4–42.**  $M_O = 4.27 \text{ N} \cdot \text{m}$ ,  $\alpha = 95.2^\circ$ ,  $\beta = 110^\circ$ ,  $\gamma = 20.6^\circ$
- 4–43.**  $\mathbf{M}_A = \{-5.39\mathbf{i} + 13.1\mathbf{j} + 11.4\mathbf{k}\} \text{ N} \cdot \text{m}$
- 4–45.**  $y = 2 \text{ m}$ ,  $z = 1 \text{ m}$
- 4–46.**  $y = 1 \text{ m}$ ,  $z = 3 \text{ m}$ ,  $d = 1.15 \text{ m}$
- 4–47.**  $\mathbf{M}_A = \{-16.0\mathbf{i} - 32.1\mathbf{k}\} \text{ N} \cdot \text{m}$
- 4–49.**  $\mathbf{M}_B = \{1.00\mathbf{i} + 0.750\mathbf{j} - 1.56\mathbf{k}\} \text{ kN} \cdot \text{m}$
- 4–50.**  $\mathbf{M}_O = \{373\mathbf{i} - 99.9\mathbf{j} + 173\mathbf{k}\} \text{ N} \cdot \text{m}$
- 4–51.**  $\theta_{\max} = 90^\circ$ ,  $\theta_{\min} = 0, 180^\circ$
- 4–53.** Yes, yes
- 4–54.**  $M_{y'} = 464 \text{ lb} \cdot \text{ft}$
- 4–55.**  $M_x = 440 \text{ lb} \cdot \text{ft}$
- 4–57.**  $\mathbf{M}_{AC} = \{11.5\mathbf{i} + 8.64\mathbf{j}\} \text{ lb} \cdot \text{ft}$
- 4–58.**  $M_x = 21.7 \text{ N} \cdot \text{m}$
- 4–59.**  $F = 139 \text{ N}$
- 4–61.**  $M_{AB} = 136 \text{ N} \cdot \text{m}$
- 4–62.**  $M_{BC} = 165 \text{ N} \cdot \text{m}$
- 4–63.**  $M_{CA} = 226 \text{ N} \cdot \text{m}$
- 4–65.**  $F = 5.66 \text{ N}$
- 4–66.**  $M_a = 4.37 \text{ N} \cdot \text{m}$ ,  $\alpha = 33.7^\circ$ ,  $\beta = 90^\circ$ ,  $\gamma = 56.3^\circ$ ,  
 $M = 5.41 \text{ N} \cdot \text{m}$
- 4–67.**  $R = 28.9 \text{ N}$
- 4–69.**  $F = 75 \text{ N}$ ,  $P = 100 \text{ N}$
- 4–70.**  $(M_R)_C = 435 \text{ lb} \cdot \text{ft} \curvearrowright$
- 4–71.**  $F = 139 \text{ lb}$
- 4–73.**  $F = 830 \text{ N}$

- 4-74.**  $M_C = 22.5 \text{ N} \cdot \text{m}$
- 4-75.**  $F = 83.3 \text{ N}$
- 4-77.**  $(M_R)_C = 240 \text{ lb} \cdot \text{ft}$
- 4-78.**  $F = 167 \text{ lb}$ . Resultant couple can act anywhere.
- 4-79.**  $d = 2.03 \text{ ft}$
- 4-81.**  $M_C = 126 \text{ lb} \cdot \text{ft}$
- 4-82.**  $\mathbf{M}_C = \{-50\mathbf{i} + 60\mathbf{j}\} \text{ lb} \cdot \text{ft}$
- 4-83.**  $M_R = 96.0 \text{ lb} \cdot \text{ft}$ ,  $\alpha = 47.4^\circ$ ,  $\beta = 74.9^\circ$ ,  $\gamma = 133^\circ$
- 4-85.**  $M_R = 64.0 \text{ lb} \cdot \text{ft}$ ,  $\alpha = 94.7^\circ$ ,  $\beta = 13.2^\circ$ ,  $\gamma = 102^\circ$
- 4-86.**  $M_2 = 424 \text{ N} \cdot \text{m}$ ,  $M_3 = 300 \text{ N} \cdot \text{m}$
- 4-87.**  $M_R = 576 \text{ lb} \cdot \text{in}$ ,  $\alpha = 37.0^\circ$ ,  $\beta = 111^\circ$ ,  $\gamma = 61.2^\circ$
- 4-89.**  $F = 15.4 \text{ N}$
- 4-90.**  $M_C = 45.1 \text{ N} \cdot \text{m}$
- 4-91.**  $F = 832 \text{ N}$
- 4-93.**  $F = 98.1 \text{ N}$
- 4-94.**  $\mathbf{M}_C = \{-2\mathbf{i} + 20\mathbf{j} + 17\mathbf{k}\} \text{ kN} \cdot \text{m}$ ,  
 $M_C = 26.3 \text{ kN} \cdot \text{m}$
- 4-95.**  $(M_C)_R = 71.9 \text{ N} \cdot \text{m}$ ,  $\alpha = 44.2^\circ$ ,  $\beta = 131^\circ$ ,  $\gamma = 103^\circ$
- 4-97.**  $F_R = 365 \text{ N}$ ,  $\theta = 70.8^\circ$ ,  $(M_R)_O = 2364 \text{ N} \cdot \text{m}$
- 4-98.**  $F_R = 365 \text{ N}$ ,  $\theta = 70.8^\circ$ ,  $(M_R)_P = 2799 \text{ N} \cdot \text{m}$
- 4-99.**  $F_R = 5.93 \text{ kN}$ ,  $\theta = 77.8^\circ$ ,  $M_{R_A} = 34.8 \text{ kN} \cdot \text{m}$
- 4-101.**  $F_R = 294 \text{ N}$ ,  $\theta = 40.1^\circ$ ,  
 $M_{R_O} = 39.6 \text{ N} \cdot \text{m}$
- 4-102.**  $F_R = 1.30 \text{ kN}$ ,  $\theta = 86.7^\circ$ ,  
 $(M_R)_A = 1.02 \text{ kN} \cdot \text{m}$
- 4-103.**  $F_R = 1.30 \text{ kN}$ ,  $\theta = 86.7^\circ$ ,  
 $(M_R)_B = 10.1 \text{ kN} \cdot \text{m}$
- 4-105.**  $F_R = 938 \text{ N}$ ,  $\theta = 35.9^\circ$ ,  $(M_R)_A = 680 \text{ N} \cdot \text{m}$
- 4-106.**  $\mathbf{M}_{R_O} = \{0.650\mathbf{i} + 19.75\mathbf{j} - 9.05\mathbf{k}\} \text{ kN} \cdot \text{m}$
- 4-107.**  $F_R = \{270\mathbf{k}\} \text{ N}$ ,  $M_{R_O} = \{-2.22\mathbf{i}\} \text{ N} \cdot \text{m}$
- 4-109.**  $F_R = \{6\mathbf{i} + 5\mathbf{j} - 5\mathbf{k}\} \text{ kN}$ ,  
 $(M_R)_O = \{2.5\mathbf{i} - 7\mathbf{j}\} \text{ kN} \cdot \text{m}$
- 4-110.**  $F_R = \{44.5\mathbf{i} + 53.1\mathbf{j} - 40.0\mathbf{k}\} \text{ N}$ ,  
 $M_{R_A} = \{-5.39\mathbf{i} + 13.1\mathbf{j} + 11.4\mathbf{k}\} \text{ N} \cdot \text{m}$
- 4-111.**  $F_R = \{-40\mathbf{j} - 40\mathbf{k}\} \text{ N}$ ,  
 $M_{R_A} = \{-12\mathbf{j} + 12\mathbf{k}\} \text{ N} \cdot \text{m}$
- 4-113.**  $F_R = 10.75 \text{ kip} \downarrow$ ,  $d = 13.7 \text{ ft}$
- 4-114.**  $F_R = 10.75 \text{ kip} \downarrow$ ,  $d = 9.26 \text{ ft}$
- 4-115.**  $F = 798 \text{ lb}$ ,  $67.9^\circ$ ,  $x = 7.43 \text{ ft}$
- 4-117.**  $F = 1302 \text{ N}$ ,  $\theta = 84.5^\circ$ ,  $x = 7.36 \text{ m}$
- 4-118.**  $F = 1302 \text{ N}$ ,  $\theta = 84.5^\circ$ ,  
 $x = 1.36 \text{ m}$  (to the right)
- 4-119.**  $F_R = 1000 \text{ N}$ ,  $\theta = 53.1^\circ$ ,  $d = 2.17 \text{ m}$
- 4-121.**  $F_R = 356 \text{ N}$ ,  $\theta = 51.8^\circ$ ,  $d = 1.75 \text{ m}$
- 4-122.**  $F_R = 542 \text{ N}$ ,  $\theta = 10.6^\circ$ ,  $d = 0.827 \text{ m}$
- 4-123.**  $F_R = 542 \text{ N}$ ,  $\theta = 10.6^\circ$ ,  $d = 2.17 \text{ m}$
- 4-125.**  $F_R = 197 \text{ lb}$ ,  $\theta = 42.6^\circ$ ,  $d = 5.24 \text{ ft}$
- 4-126.**  $F_R = 197 \text{ lb}$ ,  $\theta = 42.6^\circ$ ,  $d = 0.824 \text{ ft}$
- 4-127.**  $F_R = 26 \text{ kN}$ ,  $y = 82.7 \text{ mm}$ ,  $x = 3.85 \text{ mm}$
- 4-129.**  $F_C = 600 \text{ N}$ ,  $F_D = 500 \text{ N}$
- 4-130.**  $F_R = 35 \text{ kN}$ ,  $y = 11.3 \text{ m}$ ,  $x = 11.5 \text{ m}$
- 4-131.**  $F_1 = 27.6 \text{ kN}$ ,  $F_2 = 24.0 \text{ kN}$

- 4-133.**  $F_A = 30 \text{ kN}$ ,  $F_B = 20 \text{ kN}$ ,  $F_R = 190 \text{ kN}$
- 4-134.**  $\mathbf{F}_R = \{141\mathbf{i} + 100\mathbf{j} + 159\mathbf{k}\} \text{ N}$ ,  
 $\mathbf{M}_{R_O} = \{122\mathbf{i} - 183\mathbf{k}\} \text{ N} \cdot \text{m}$
- 4-135.**  $F_R = 379 \text{ N}$ ,  $M_R = 590 \text{ N} \cdot \text{m}$ ,  $z = 2.68 \text{ m}$ ,  
 $x = -2.76 \text{ m}$
- 4-137.**  $F_R = 539 \text{ N}$ ,  $M_R = 1.45 \text{ kN} \cdot \text{m}$ ,  $x = 1.21 \text{ m}$ ,  
 $y = 3.59 \text{ m}$
- 4-138.**  $F_R = 0$ ,  $M_{R_O} = 1.35 \text{ kip} \cdot \text{ft}$
- 4-139.**  $F_R = 6.75 \text{ kN}$ ,  $\bar{x} = 2.5 \text{ m}$
- 4-141.**  $F_R = 7 \text{ lb}$ ,  $\bar{x} = 0.268 \text{ ft}$
- 4-142.**  $F_R = 15.0 \text{ kN}$ ,  $d = 3.40 \text{ m}$
- 4-143.**  $F_R = 12.5 \text{ kN}$ ,  $d = 1.54 \text{ m}$
- 4-145.**  $F_R = 15.4 \text{ kN}$ ,  $(M_R)_O = 18.5 \text{ kN} \cdot \text{m}$
- 4-146.**  $F_R = 27.0 \text{ kN}$ ,  $(M_R)_A = 81.0 \text{ kN} \cdot \text{m}$
- 4-147.**  $a = 1.54 \text{ m}$
- 4-149.**  $w_2 = 17.2 \text{ kN/m}$ ,  $w_1 = 30.3 \text{ kN/m}$
- 4-150.**  $F_R = 51.0 \text{ kN} \downarrow$ ,  $M_{R_O} = 914 \text{ kN} \cdot \text{m}$
- 4-151.**  $F_R = 51.0 \text{ kN} \downarrow$ ,  $d = 17.9 \text{ m}$
- 4-153.**  $F_R = 1.80 \text{ kN}$ ,  $d = 2.33 \text{ m}$
- 4-154.**  $F_R = 12.0 \text{ kN}$ ,  $\theta = 48.4^\circ$ ,  $d = 3.28 \text{ m}$
- 4-155.**  $F_R = 12.0 \text{ kN}$ ,  $\theta = 48.4^\circ$ ,  $d = 3.69 \text{ m}$
- 4-157.**  $F_R = 6.75 \text{ kN}$ ,  $(M_R)_O = 4.05 \text{ kN} \cdot \text{m}$
- 4-158.**  $F_R = 43.6 \text{ lb}$ ,  $x = 3.27 \text{ ft}$
- 4-159.**  $d = 2.22 \text{ ft}$
- 4-161.**  $F_R = \frac{2Lw_0}{\pi}$ ,  $(M_R)_O = \left(\frac{2\pi - 4}{\pi^2}\right)w_0L^2$
- 4-162.**  $F_R = 107 \text{ kN}$ ,  $h = 1.60 \text{ m}$

## Chapter 5

- 5-10.**  $A_x = 3.46 \text{ kN}$ ,  $A_y = 8 \text{ kN}$ ,  $M_A = 20.2 \text{ kN} \cdot \text{m}$
- 5-11.**  $N_A = 750 \text{ N}$ ,  $B_y = 600 \text{ N}$ ,  $B_x = 450 \text{ N}$
- 5-13.**  $N_A = 2.175 \text{ kN}$ ,  $B_y = 1.875 \text{ kN}$ ,  $B_x = 0$
- 5-14.**  $N_A = 3.33 \text{ kN}$ ,  $B_x = 2.40 \text{ kN}$ ,  $B_y = 133 \text{ N}$
- 5-15.**  $A_y = 5.00 \text{ kN}$ ,  $N_B = 9.00 \text{ kN}$ ,  $A_x = 5.00 \text{ kN}$
- 5-17.**  $\theta = 41.4^\circ$
- 5-18.**  $A_x = 0$ ,  $B_y = P$ ,  $M_A = \frac{PL}{2}$
- 5-19.**  $T = \frac{W}{2} \sin \theta$
- 5-21.**  $T_{BC} = 113 \text{ N}$
- 5-22.**  $N_A = 3.71 \text{ kN}$ ,  $B_x = 1.86 \text{ kN}$ ,  $B_y = 8.78 \text{ kN}$
- 5-23.**  $w = 2.67 \text{ kN/m}$
- 5-25.**  $N_A = 39.7 \text{ lb}$ ,  $N_B = 82.5 \text{ lb}$ ,  $M_A = 106 \text{ lb} \cdot \text{ft}$
- 5-26.**  $\theta = 70.3^\circ$ ,  $N'_A = (29.4 - 31.3 \sin \theta) \text{ kN}$ ,  
 $N'_B = (73.6 + 31.3 \sin \theta) \text{ kN}$
- 5-27.**  $N_B = 98.1 \text{ N}$ ,  $A_x = 85.0 \text{ N}$ ,  $A_y = 147 \text{ N}$
- 5-29.**  $P = 272 \text{ N}$
- 5-30.**  $P_{\min} = 271 \text{ N}$
- 5-31.**  $F_B = 86.6 \text{ N}$ ,  $B_x = 43.3 \text{ N}$ ,  $B_y = 110 \text{ N}$
- 5-33.**  $A_x = 25.4 \text{ kN}$ ,  $B_y = 22.8 \text{ kN}$ ,  $B_x = 25.4 \text{ kN}$
- 5-34.**  $F = 14.0 \text{ kN}$

- 5-35.**  $N_A = 173 \text{ N}$ ,  $N_C = 416 \text{ N}$ ,  $N_B = 69.2 \text{ N}$
- 5-37.**  $N_A = 975 \text{ lb}$ ,  $B_x = 975 \text{ lb}$ ,  $B_y = 780 \text{ lb}$
- 5-38.**  $A_x = 1.46 \text{ kip}$ ,  $F_B = 1.66 \text{ kip}$
- 5-39.**  $\theta = 17.5^\circ$
- 5-41.**  $F = 311 \text{ kN}$ ,  $A_x = 460 \text{ kN}$ ,  $A_y = 7.85 \text{ kN}$
- 5-42.**  $F_{CB} = 782 \text{ N}$ ,  $A_x = 625 \text{ N}$ ,  $A_y = 681 \text{ N}$
- 5-43.**  $F_2 = 724 \text{ N}$ ,  $F_1 = 1.45 \text{ kN}$ ,  $F_A = 1.75 \text{ kN}$
- 5-45.**  $P = 660 \text{ N}$ ,  $N_A = 442 \text{ N}$ ,  $\theta = 48.0^\circ$
- 5-46.**  $d = \frac{3a}{4}$
- 5-47.**  $F_{BC} = 80 \text{ kN}$ ,  $A_x = 54 \text{ kN}$ ,  $A_y = 16 \text{ kN}$
- 5-49.**  $F_C = 10 \text{ mN}$
- 5-50.**  $k = 250 \text{ N/m}$
- 5-51.**  $w_B = 2.19 \text{ kip/ft}$ ,  $w_A = 10.7 \text{ kip/ft}$
- 5-53.**  $\alpha = 10.4^\circ$
- 5-54.**  $h = 0.645 \text{ m}$
- 5-55.**  $h = \sqrt{\frac{s^2 - l^2}{3}}$
- 5-57.**  $w_1 = 83.3 \text{ lb/ft}$ ,  $w_2 = 167 \text{ lb/ft}$
- 5-58.**  $w_1 = \frac{2P}{L}$ ,  $w_2 = \frac{4P}{L}$
- 5-59.**  $\theta = 23.2^\circ$ ,  $85.2^\circ$
- 5-61.**  $N_A = 346 \text{ N}$ ,  $N_B = 693 \text{ N}$ ,  $a = 0.650 \text{ m}$
- 5-62.**  $T = 1.84 \text{ kN}$ ,  $F = 6.18 \text{ kN}$
- 5-63.**  $R_D = 22.6 \text{ kip}$ ,  $R_E = 22.6 \text{ kip}$ ,  $R_F = 13.7 \text{ kip}$
- 5-65.**  $N_A = 28.6 \text{ lb}$ ,  $N_B = 10.7 \text{ lb}$ ,  $N_C = 10.7 \text{ lb}$
- 5-66.**  $T_{BC} = 43.9 \text{ N}$ ,  $N_B = 58.9 \text{ N}$ ,  $A_x = 58.9 \text{ N}$ ,  $A_y = 39.2 \text{ N}$ ,  $A_z = 177 \text{ N}$
- 5-67.**  $T_C = 14.8 \text{ kN}$ ,  $T_B = 16.5 \text{ kN}$ ,  $T_A = 7.27 \text{ kN}$
- 5-69.**  $F_{AB} = 467 \text{ N}$ ,  $F_{AC} = 674 \text{ N}$ ,  $D_x = 1.04 \text{ kN}$ ,  $D_y = 0$ ,  $D_z = 0$
- 5-70.**  $T_{BA} = 2.00 \text{ kN}$ ,  $T_{BC} = 1.35 \text{ kN}$ ,  $D_x = 0.327 \text{ kN}$ ,  $D_y = 1.31 \text{ kN}$ ,  $D_z = 4.58 \text{ kN}$
- 5-71.**  $F_{BD} = F_{BC} = 350 \text{ N}$ ,  $A_x = 600 \text{ N}$ ,  $A_y = 0$ ,  $A_z = 300 \text{ N}$
- 5-73.**  $C_y = 800 \text{ N}$ ,  $B_z = 107 \text{ N}$ ,  $B_y = 600 \text{ N}$ ,  $C_x = 53.6 \text{ N}$ ,  $A_x = 400 \text{ N}$ ,  $A_z = 800 \text{ N}$
- 5-74.**  $F = 746 \text{ N}$
- 5-75.**  $T_{BC} = 1.40 \text{ kN}$ ,  $A_y = 800 \text{ N}$ ,  $A_x = 1.20 \text{ kN}$ ,  $(M_A)_x = 600 \text{ N}\cdot\text{m}$ ,  $(M_A)_y = 1.20 \text{ kN}\cdot\text{m}$ ,  $(M_A)_z = 2.40 \text{ kN}\cdot\text{m}$
- 5-77.**  $A_x = 300 \text{ N}$ ,  $A_y = 500 \text{ N}$ ,  $N_B = 400 \text{ N}$ ,  $(M_A)_x = 1.00 \text{ kN}\cdot\text{m}$ ,  $(M_A)_y = 200 \text{ N}\cdot\text{m}$ ,  $(M_A)_z = 1.50 \text{ kN}\cdot\text{m}$
- 5-78.**  $A_x = 633 \text{ lb}$ ,  $A_y = -141 \text{ lb}$ ,  $B_x = -721 \text{ lb}$ ,  $B_z = 895 \text{ lb}$ ,  $C_y = 200 \text{ lb}$ ,  $C_z = -506 \text{ lb}$
- 5-79.**  $F_2 = 674 \text{ lb}$
- 5-81.**  $C_z = 10.6 \text{ lb}$ ,  $D_y = -0.230 \text{ lb}$ ,  $C_y = 0.230 \text{ lb}$ ,  $D_x = 5.17 \text{ lb}$ ,  $C_x = 5.44 \text{ lb}$ ,  $M = 0.459 \text{ lb}\cdot\text{ft}$
- 5-82.**  $F_{BD} = 294 \text{ N}$ ,  $F_{BC} = 589 \text{ N}$ ,  $A_x = 0$ ,  $A_y = 589 \text{ N}$ ,  $A_z = 490.5 \text{ N}$

- 5-83.**  $T = 58.0 \text{ N}$ ,  $C_z = 87.0 \text{ N}$ ,  $C_y = 28.8 \text{ N}$ ,  $D_x = 0$ ,  $D_y = 79.2 \text{ N}$ ,  $D_z = 58.0 \text{ N}$
- 5-85.**  $F_{BC} = 0$ ,  $A_y = 0$ ,  $A_z = 800 \text{ lb}$ ,  $(M_A)_x = 4.80 \text{ kip}\cdot\text{ft}$ ,  $(M_A)_y = 0$ ,  $(M_A)_z = 0$

## Chapter 6

- 6-1.**  $F_{CB} = 0$ ,  $F_{CD} = 20.0 \text{ kN}$  (C),  $F_{DB} = 33.3 \text{ kN}$  (T),  $F_{DA} = 36.7 \text{ kN}$  (C)
- 6-2.**  $F_{CB} = 0$ ,  $F_{CD} = 45.0 \text{ kN}$  (C),  $F_{DB} = 75.0 \text{ kN}$  (T),  $F_{DA} = 90.0 \text{ kN}$  (C)
- 6-3.**  $F_{AC} = 150 \text{ lb}$  (C),  $F_{AB} = 140 \text{ lb}$  (T),  $F_{BD} = 140 \text{ lb}$  (T),  $F_{BC} = 0$ ,  $F_{CD} = 150 \text{ lb}$  (T),  $F_{CE} = 180 \text{ lb}$  (C),  $F_{DE} = 120 \text{ lb}$  (C),  $F_{DF} = 230 \text{ lb}$  (T),  $F_{EF} = 300 \text{ lb}$  (C)
- 6-5.**  $F_{CD} = 5.21 \text{ kN}$  (C),  $F_{CB} = 4.17 \text{ kN}$  (T),  $F_{AD} = 1.46 \text{ kN}$  (C),  $F_{AB} = 4.17 \text{ kN}$  (T),  $F_{BD} = 4 \text{ kN}$  (T)
- 6-6.**  $F_{CD} = 5.21 \text{ kN}$  (C),  $F_{CB} = 2.36 \text{ kN}$  (T),  $F_{AD} = 1.46 \text{ kN}$  (C),  $F_{AB} = 2.36 \text{ kN}$  (T),  $F_{BD} = 4 \text{ kN}$  (T)
- 6-7.**  $F_{DE} = 16.3 \text{ kN}$  (C),  $F_{DC} = 8.40 \text{ kN}$  (T),  $F_{EA} = 8.85 \text{ kN}$  (C),  $F_{EC} = 6.20 \text{ kN}$  (C),  $F_{CF} = 8.77 \text{ kN}$  (T),  $F_{CB} = 2.20 \text{ kN}$  (T),  $F_{BA} = 3.11 \text{ kN}$  (T),  $F_{BF} = 6.20 \text{ kN}$  (C),  $F_{FA} = 6.20 \text{ kN}$  (T)
- 6-9.**  $F_{AE} = 5.66 \text{ kN}$  (C),  $F_{AB} = 4.00 \text{ kN}$  (T),  $F_{DE} = 7.07 \text{ kN}$  (C),  $F_{DC} = 5.00 \text{ kN}$  (T),  $F_{BE} = 3.16 \text{ kN}$  (T),  $F_{BC} = 3.00 \text{ kN}$  (T),  $F_{CE} = 6.32 \text{ kN}$  (T)
- 6-10.**  $F_{AE} = 9.90 \text{ kN}$  (C),  $F_{AB} = 7.00 \text{ kN}$  (T),  $F_{DE} = 11.3 \text{ kN}$  (C),  $F_{DC} = 8.00 \text{ kN}$  (T),  $F_{BE} = 6.32 \text{ kN}$  (T),  $F_{BC} = 5.00 \text{ kN}$  (T),  $F_{CE} = 9.49 \text{ kN}$  (T)
- 6-11.**  $F_{JD} = 33.3 \text{ kN}$  (T),  $F_{AL} = F_{GH} = F_{LK} = F_{HI} = 28.3 \text{ kN}$  (C),  $F_{AB} = F_{GF} = F_{BC} = F_{FE} = F_{CD} = F_{ED} = 20 \text{ kN}$  (T),  $F_{BL} = F_{FH} = F_{LC} = F_{HE} = 0$ ,  $F_{CK} = F_{EI} = 10 \text{ kN}$  (T),  $F_{KJ} = F_{IJ} = 23.6 \text{ kN}$  (C),  $F_{KD} = F_{ID} = 7.45 \text{ kN}$  (C)
- 6-13.**  $F_{CD} = F_{AD} = 0.687P$  (T),  $F_{CB} = F_{AB} = 0.943P$  (C),  $F_{DB} = 1.33P$  (T)
- 6-14.**  $P_{\max} = 849 \text{ lb}$
- 6-15.**  $P_{\max} = 849 \text{ lb}$
- 6-17.**  $P = 5.20 \text{ kN}$
- 6-18.**  $F_{DE} = 8.94 \text{ kN}$  (T),  $F_{DC} = 4.00 \text{ kN}$  (C),  $F_{CB} = 4.00 \text{ kN}$  (C),  $F_{CE} = 0$ ,  $F_{EB} = 11.3 \text{ kN}$  (C),  $F_{EF} = 12.0 \text{ kN}$  (T),  $F_{BA} = 12.0 \text{ kN}$  (C),  $F_{BF} = 18.0 \text{ kN}$  (T),  $F_{FA} = 20.1 \text{ kN}$  (C),  $F_{FG} = 21.0 \text{ kN}$  (T)

- 6–19.**  $F_{DE} = 13.4 \text{ kN (T)}, F_{DC} = 6.00 \text{ kN (C)},$   
 $F_{CB} = 6.00 \text{ kN (C)}, F_{CE} = 0, F_{EB} = 17.0 \text{ kN (C)},$   
 $F_{EF} = 18.0 \text{ kN (T)}, F_{BA} = 18.0 \text{ kN (C)},$   
 $F_{BF} = 20.0 \text{ kN (T)}, F_{FA} = 22.4 \text{ kN (C)},$   
 $F_{FG} = 28.0 \text{ kN (T)}$
- 6–21.**  $F_{DE} = F_{DC} = F_{FA} = 0, F_{CE} = 34.4 \text{ kN (C)},$   
 $F_{CB} = 20.6 \text{ kN (T)}, F_{BA} = 20.6 \text{ kN (T)},$   
 $F_{BE} = 15.0 \text{ kN (T)}, F_{FE} = 30.0 \text{ kN (C)},$   
 $F_{EA} = 15.6 \text{ kN (T)}$
- 6–22.**  $F_{FE} = 0.667P \text{ (T)}, F_{FD} = 1.67P \text{ (T)},$   
 $F_{AB} = 0.471P \text{ (C)}, F_{AE} = 1.67P \text{ (T)},$   
 $F_{AC} = 1.49P \text{ (C)}, F_{BF} = 1.41P \text{ (T)},$   
 $F_{BD} = 1.49P \text{ (C)}, F_{EC} = 1.41P \text{ (T)},$   
 $F_{CD} = 0.471P \text{ (C)}$
- 6–23.**  $F_{EC} = 1.20P \text{ (T)}, F_{ED} = 0,$   
 $F_{AB} = F_{AD} = 0.373P \text{ (C)}, F_{DC} = 0.373P \text{ (C)},$   
 $F_{DB} = 0.333P \text{ (T)}, F_{BC} = 0.373P \text{ (C)}$
- 6–25.**  $F_{CB} = 2.31 \text{ kN (C)}, F_{CD} = 1.15 \text{ kN (C)},$   
 $F_{DB} = 4.00 \text{ kN (T)}, F_{DA} = 4.62 \text{ kN (C)},$   
 $F_{AB} = 2.31 \text{ kN (C)}$
- 6–26.**  $P_{\max} = 1.30 \text{ kN}$
- 6–27.**  $F_{HI} = 42.5 \text{ kN (T)}, F_{HC} = 100 \text{ kN (T)},$   
 $F_{DC} = 125 \text{ kN (C)}$
- 6–29.**  $F_{HG} = 1125 \text{ lb (T)}, F_{DE} = 3375 \text{ lb (C)},$   
 $F_{EH} = 3750 \text{ lb (T)}$
- 6–30.**  $F_{CD} = 3375 \text{ lb (C)}, F_{HI} = 6750 \text{ lb (T)},$   
 $F_{CH} = 5625 \text{ lb (C)}$
- 6–31.**  $F_{KJ} = 11.25 \text{ kip (T)}, F_{CD} = 9.375 \text{ kip (C)},$   
 $F_{CJ} = 3.125 \text{ kip (C)}, F_{DJ} = 0$
- 6–33.**  $F_{GF} = 12.5 \text{ kN (C)}, F_{CD} = 6.67 \text{ kN (T)}, F_{GC} = 0$
- 6–34.**  $F_{GH} = 12.5 \text{ kN (C)}, F_{BG} = 6.01 \text{ kN (T)},$   
 $F_{BC} = 6.67 \text{ kN (T)}$
- 6–35.**  $F_{BC} = 5.33 \text{ kN (C)}, F_{EF} = 5.33 \text{ kN (T)},$   
 $F_{CF} = 4.00 \text{ kN (T)}$
- 6–37.**  $F_{EF} = 14.0 \text{ kN (C)}, F_{BC} = 13.0 \text{ kN (T)},$   
 $F_{BE} = 1.41 \text{ kN (T)}, F_{BF} = 8.00 \text{ kN (T)}$
- 6–38.**  $F_{EF} = 15.0 \text{ kN (C)}, F_{BC} = 12.0 \text{ kN (T)},$   
 $F_{BE} = 4.24 \text{ kN (T)}$
- 6–39.**  $F_{BC} = 10.4 \text{ kN (C)}, F_{HG} = 9.16 \text{ kN (T)},$   
 $F_{HC} = 2.24 \text{ kN (T)}$
- 6–41.**  $F_{BC} = 18.0 \text{ kN (T)}, F_{FE} = 15.0 \text{ kN (C)},$   
 $F_{EB} = 5.00 \text{ kN (C)}$
- 6–42.**  $F_{HG} = 17.6 \text{ kN (C)}, F_{HC} = 5.41 \text{ kN (C)},$   
 $F_{BC} = 19.1 \text{ kN (T)}$
- 6–43.**  $F_{GJ} = 17.6 \text{ kN (C)}, F_{CJ} = 8.11 \text{ kN (C)},$   
 $F_{CD} = 21.4 \text{ kN (T)}, F_{CG} = 7.50 \text{ kN (T)}$
- 6–45.**  $F_{BF} = 0, F_{BG} = 35.4 \text{ kN (C)}, F_{AB} = 45 \text{ kN (T)}$
- 6–46.**  $F_{BC} = 11.0 \text{ kN (T)}, F_{GH} = 11.2 \text{ kN (C)},$   
 $F_{CH} = 1.25 \text{ kN (C)}, F_{CG} = 10.0 \text{ kN (T)}$
- 6–47.**  $F_{CD} = 18.0 \text{ kN (T)}, F_{CJ} = 10.8 \text{ kN (T)},$   
 $F_{KJ} = 26.8 \text{ kN (T)}$
- 6–49.**  $F_{EF} = 12.9 \text{ kN (T)}, F_{FI} = 7.21 \text{ kN (T)},$   
 $F_{HI} = 21.1 \text{ kN (C)}$
- 6–50.**  $F_{CA} = F_{CB} = 122 \text{ lb (C)}, F_{CD} = 173 \text{ lb (T)},$   
 $F_{BD} = 86.6 \text{ lb (T)}, F_{BA} = 0, F_{DA} = 86.6 \text{ lb (T)}$
- 6–51.**  $F_{AB} = 6.46 \text{ kN (T)}, F_{AC} = F_{AD} = 1.50 \text{ kN (C)},$   
 $F_{BC} = F_{BD} = 3.70 \text{ kN (C)}, F_{BE} = 4.80 \text{ kN (T)}$
- 6–53.**  $F_{CA} = 833 \text{ lb (T)}, F_{CB} = 667 \text{ lb (C)},$   
 $F_{CD} = 333 \text{ lb (T)}, F_{AD} = F_{AB} = 354 \text{ lb (C)},$   
 $F_{DB} = 50 \text{ lb (T)}$
- 6–54.**  $F_{CA} = 1000 \text{ lb (C)}, F_{CD} = 406 \text{ lb (T)},$   
 $F_{CB} = 344 \text{ lb (C)}, F_{AB} = F_{AD} = 424 \text{ lb (T)},$   
 $F_{DB} = 544 \text{ lb (C)}$
- 6–55.**  $F_{DF} = 5.31 \text{ kN (C)}, F_{EF} = 2.00 \text{ kN (T)},$   
 $F_{AF} = 0.691 \text{ kN (T)}$
- 6–57.**  $F_{BF} = 0, F_{BC} = 0, F_{BE} = 500 \text{ lb (T)},$   
 $F_{AB} = 300 \text{ lb (C)}, F_{AC} = 583 \text{ lb (T)},$   
 $F_{AD} = 333 \text{ lb (T)}, F_{AE} = 667 \text{ lb (C)}, F_{DE} = 0,$   
 $F_{EF} = 300 \text{ lb (C)}, F_{CD} = 300 \text{ lb (C)},$   
 $F_{CF} = 300 \text{ lb (C)}, F_{DF} = 424 \text{ lb (T)}$
- 6–58.**  $F_{BF} = 0, F_{BC} = 0, F_{BE} = 500 \text{ lb (T)},$   
 $F_{AB} = 300 \text{ lb (C)}, F_{AC} = 972 \text{ lb (T)}, F_{AD} = 0,$   
 $F_{AE} = 367 \text{ lb (C)}, F_{DE} = 0, F_{EF} = 300 \text{ lb (C)},$   
 $F_{CD} = 500 \text{ lb (C)}, F_{CF} = 300 \text{ lb (C)},$   
 $F_{DF} = 424 \text{ lb (T)}$
- 6–59.**  $F_{AD} = 686 \text{ N (T)}, F_{BD} = 0, F_{CD} = 615 \text{ N (C)},$   
 $F_{BC} = 229 \text{ N (T)}, F_{AC} = 343 \text{ N (T)},$   
 $F_{EC} = 457 \text{ N (C)}$
- 6–61.**  $P = 12.5 \text{ lb}$
- 6–62.** **a.**  $P = 25.0 \text{ lb}$ , **b.**  $P = 33.3 \text{ lb}$ , **c.**  $P = 11.1 \text{ lb}$
- 6–63.**  $P = 18.9 \text{ N}$
- 6–65.**  $B_x = 4.00 \text{ kN}, B_y = 5.33 \text{ kN}, A_x = 4.00 \text{ kN},$   
 $A_y = 5.33 \text{ kN}$
- 6–66.**  $A_x = 24.0 \text{ kN}, A_y = 12.0 \text{ kN}, D_x = 18.0 \text{ kN},$   
 $D_y = 24.0 \text{ kN}$
- 6–67.**  $A_x = 120 \text{ lb}, A_y = 0, N_C = 15.0 \text{ lb}$
- 6–69.**  $B_x = 2.80 \text{ kip}, B_y = 1.05 \text{ kip}, A_x = 2.80 \text{ kip},$   
 $A_y = 5.10 \text{ kip}, M_A = 43.2 \text{ kip} \cdot \text{ft}$
- 6–70.**  $C_y = 184 \text{ N}, C_x = 490.5 \text{ N}, B_x = 1.23 \text{ kN},$   
 $B_y = 920 \text{ kN}$
- 6–71.**  $N_E = 18.0 \text{ kN}, N_C = 4.50 \text{ kN}, A_x = 0,$   
 $A_y = 7.50 \text{ kN}, M_A = 22.5 \text{ kN} \cdot \text{m}$
- 6–73.**  $N_E = 3.60 \text{ kN}, N_B = 900 \text{ N}, A_x = 0,$   
 $A_y = 2.70 \text{ kN}, M_A = 8.10 \text{ kN} \cdot \text{m}$
- 6–74.**  $T = 350 \text{ lb}, A_y = 700 \text{ lb}, A_x = 1.88 \text{ kip},$   
 $D_x = 1.70 \text{ kip}, D_y = 1.70 \text{ kip}$
- 6–75.**  $T = 350 \text{ lb}, A_y = 700 \text{ lb}, D_x = 1.82 \text{ kip},$   
 $D_y = 1.84 \text{ kip}, A_x = 2.00 \text{ kip}$
- 6–77.**  $A_x = 96 \text{ lb}, A_y = 72 \text{ lb}, D_y = 2.18 \text{ kip},$   
 $E_x = 96.0 \text{ lb}, E_y = 1.61 \text{ kip}$
- 6–78.**  $N_C = 3.00 \text{ kN}, N_A = 3.00 \text{ kN},$   
 $B_y = 18.0 \text{ kN}, B_x = 0$

- 6-79.**  $N_C = N_D = 2 \text{ lb}$   
**6-81.**  $F_{FB} = 1.94 \text{ kN}, F_{BD} = 2.60 \text{ kN}$   
**6-82.**  $N_A = 36.0 \text{ lb}$   
**6-83.**  $F_{FD} = 20.1 \text{ kN}, F_{BD} = 25.5 \text{ kN}$   
 Member *EDC*:  $C'_x = 18.0 \text{ kN}, C'_y = 12.0 \text{ kN}$ ,  
 Member *ABC*:  $C''_y = 12.0 \text{ kN}, C''_x = 18.0 \text{ kN}$   
**6-85.**  $T_{AI} = 2.88 \text{ kip}, F_H = 3.99 \text{ kip}$   
**6-86.**  $M = 314 \text{ lb} \cdot \text{ft}$   
**6-87.**  $F_C = 19.6 \text{ kN}$   
**6-89.**  $C_x = 650 \text{ N}, C_y = 0$   
**6-90.**  $N_B = N_C = 49.5 \text{ N}$   
**6-91.**  $F_{EF} = 8.18 \text{ kN (T)}, F_{AD} = 158 \text{ kN (C)}$
- 6-93.**  $P(\theta) = \frac{250 \sqrt{2.25^2 - \cos^2 \theta}}{\sin \theta \cos \theta + \sqrt{2.25^2 - \cos^2 \theta} \cdot \cos \theta}$
- 6-94.**  $N_B = 0.1175 \text{ lb}, N_A = 0.0705 \text{ lb}$   
**6-95.**  $F_N = 5.25 \text{ lb}$   
**6-97.** a.  $F = 205 \text{ lb}, N_C = 380 \text{ lb}$ ,  
 b.  $F = 102 \text{ lb}, N_C = 72.5 \text{ lb}$   
**6-98.**  $E_y = 1.00 \text{ kN}, E_x = 3.00 \text{ kN}, B_x = 2.50 \text{ kN}$ ,  
 $B_y = 1.00 \text{ kN}, A_x = 2.50 \text{ kN}, A_y = 500 \text{ N}$   
**6-99.**  $N_C = 12.7 \text{ kN}, A_x = 12.7 \text{ kN}, A_y = 2.94 \text{ kN}$ ,  
 $N_D = 1.05 \text{ kN}$   
**6-101.**  $F = 370 \text{ N}$   
**6-102.**  $N_A = 284 \text{ N}$   
**6-103.**  $B_y = 2.67 \text{ kN}, B_x = 4.25 \text{ kN}$ ,  
 $A_y = 3.33 \text{ kN}, A_x = 7.25 \text{ kN}$   
**6-105.**  $P = 198 \text{ N}$   
**6-106.**  $F = 66.1 \text{ lb}$   
**6-107.**  $d = 0.638 \text{ ft}$   
**6-109.**  $P = 46.9 \text{ lb}$   
**6-110.**  $\theta = 23.7^\circ$   
**6-111.**  $m = 26.0 \text{ kg}$   
**6-113.**  $m_S = 1.71 \text{ kg}$   
**6-114.**  $m_L = 106 \text{ kg}$   
**6-115.**  $P = 283 \text{ N}, B_x = D_x = 42.5 \text{ N}$ ,  
 $B_y = D_y = 283 \text{ N}, B_z = D_z = 283 \text{ N}$   
**6-117.**  $M_{Ex} = 0.5 \text{ kN} \cdot \text{m}, M_{Ey} = 0, E_y = 0, E_x = 0$   
**6-118.**  $F_D = 20.8 \text{ lb}, F_F = 14.7 \text{ lb}, F_A = 24.5 \text{ lb}$

**Chapter 7**

- 7-1.**  $N_C = 0, V_C = -386 \text{ lb}, M_C = -857 \text{ lb} \cdot \text{ft}$ ,  
 $N_D = 0, V_D = 300 \text{ lb}, M_D = -600 \text{ lb} \cdot \text{ft}$   
**7-2.**  $N_C = 0, V_C = -1.00 \text{ kip}, M_C = 56.0 \text{ kip} \cdot \text{ft}$ ,  
 $N_D = 0, V_D = -1.00 \text{ kip}, M_D = 48.0 \text{ kip} \cdot \text{ft}$   
**7-3.**  $V_A = 0, N_A = -39 \text{ kN}, M_A = -2.425 \text{ kN} \cdot \text{m}$   
**7-5.**  $V_C = -133 \text{ lb}, M_C = 133 \text{ lb} \cdot \text{in.}$   
**7-6.**  $a = \frac{L}{3}$   
**7-7.**  $V_C = -4.00 \text{ kip}, M_C = 24.0 \text{ kip} \cdot \text{ft}$   
**7-9.**  $N_C = -30 \text{ kN}, V_C = -8 \text{ kN}, M_C = 6 \text{ kN} \cdot \text{m}$

- 7-10.**  $P = 0.533 \text{ kN}, N_C = -2 \text{ kN}, V_C = -0.533 \text{ kN}$ ,  
 $M_C = 0.400 \text{ kN} \cdot \text{m}$   
**7-11.**  $N_C = 265 \text{ lb}, V_C = -649 \text{ lb}, M_C = -4.23 \text{ kip} \cdot \text{ft}$ ,  
 $N_D = -265 \text{ lb}, V_D = 637 \text{ lb}, M_D = -3.18 \text{ kip} \cdot \text{ft}$   
**7-13.**  $N_D = 0, V_D = 3.00 \text{ kip}, M_D = 12.0 \text{ kip} \cdot \text{ft}$ ,  
 $N_E = 0, V_E = -8.00 \text{ kip}, M_E = -20.0 \text{ kip} \cdot \text{ft}$   
**7-14.**  $M_C = -15.0 \text{ kip} \cdot \text{ft}, N_C = 0, V_C = 2.01 \text{ kip}$ ,  
 $M_D = 3.77 \text{ kip} \cdot \text{ft}, N_D = 0, V_D = 1.11 \text{ kip}$   
**7-15.**  $N_C = 0, V_C = -1.50 \text{ kN}, M_C = 13.5 \text{ kN} \cdot \text{m}$   
**7-17.**  $N_A = 86.6 \text{ lb}, V_A = 150 \text{ lb}, M_A = 1.80 \text{ kip} \cdot \text{in.}$   
**7-18.**  $V_C = 2.49 \text{ kN}, N_C = 2.49 \text{ kN}, M_C = 4.97 \text{ kN} \cdot \text{m}$ ,  
 $N_D = 0, V_D = -2.49 \text{ kN}, M_D = 16.5 \text{ kN} \cdot \text{m}$   
**7-19.**  $N_C = -4.32 \text{ kip}, V_C = 1.35 \text{ kip}, M_C = 4.72 \text{ kip} \cdot \text{ft}$   
**7-21.**  $N_E = 720 \text{ N}, V_E = 1.12 \text{ kN}, M_E = -320 \text{ N} \cdot \text{m}$ ,  
 $N_F = 0, V_F = -1.24 \text{ kN}, M_F = -1.41 \text{ kN} \cdot \text{m}$   
**7-22.**  $N_D = 4 \text{ kN}, V_D = -9 \text{ kN}, M_D = -18 \text{ kN} \cdot \text{m}$ ,  
 $N_E = 4 \text{ kN}, V_E = 3.75 \text{ kN}, M_E = -4.875 \text{ kN} \cdot \text{m}$   
**7-23.**  $N_C = 400 \text{ N}, V_C = -96 \text{ N}, M_C = -144 \text{ N} \cdot \text{m}$   
**7-25.**  $N_D = 0, V_D = 0.75 \text{ kip}, M_D = 13.5 \text{ kip} \cdot \text{ft}$ ,  
 $N_E = 0, V_E = -9 \text{ kip}, M_E = -24.0 \text{ kip} \cdot \text{ft}$   
**7-26.**  $N_C = -20.0 \text{ kN}, V_C = 70.6 \text{ kN}$ ,  
 $M_C = -302 \text{ kN} \cdot \text{m}$   
**7-27.**  $N_C = -1.60 \text{ kN}, V_C = 200 \text{ N}, M_C = 200 \text{ N} \cdot \text{m}$   
**7-29.**  $N_C = -406 \text{ lb}, V_C = 903 \text{ lb}, M_C = 1.35 \text{ kip} \cdot \text{ft}$   
**7-30.**  $N_D = -464 \text{ lb}, V_D = -203 \text{ lb}, M_D = 2.61 \text{ kip} \cdot \text{ft}$   
**7-31.**  $N_E = 2.20 \text{ kN}, V_E = 0, M_E = 0$ ,  
 $N_D = -2.20 \text{ kN}, V_D = 600 \text{ N}, M_D = 1.20 \text{ kN} \cdot \text{m}$   
**7-33.**  $N_D = -2.25 \text{ kN}, V_D = 1.25 \text{ kN}, -1.88 \text{ kN} \cdot \text{m}$   
**7-34.**  $N_E = 1.25 \text{ kN}, V_E = 0, M_B = 1.69 \text{ kN} \cdot \text{m}$   
**7-35.**  $d = 0.200 \text{ m}$   
**7-37.**  $N_D = 1.26 \text{ kN}, V_D = 0, M_D = 500 \text{ N} \cdot \text{m}$   
**7-38.**  $N_E = -1.48 \text{ kN}, V_E = 500 \text{ N}, M_E = 1000 \text{ N} \cdot \text{m}$   
**7-39.**  $V = 0.278 w_0 r, N = 0.0759 w_0 r$ ,  
 $M = 0.0759 w_0 r^2$   
**7-41.**  $N_C = -350 \text{ lb}, (V_C)_y = 700 \text{ lb}, (V_C)_z = -150 \text{ lb}$ ,  
 $(M_C)_x = -1.20 \text{ kip} \cdot \text{ft}, (M_C)_y = -750 \text{ lb} \cdot \text{ft}$ ,  
 $(M_C)_z = 1.40 \text{ kip} \cdot \text{ft}$   
**7-42.**  $(V_C)_x = 104 \text{ lb}, N_C = 0, (V_C)_z = 10 \text{ lb}$ ,  
 $(M_C)_x = 20 \text{ lb} \cdot \text{ft}, (M_C)_y = 72 \text{ lb} \cdot \text{ft}$ ,  
 $(M_C)_z = -178 \text{ lb} \cdot \text{ft}$   
**7-43.**  $N_x = -500 \text{ N}, V_y = 100 \text{ N}, V_z = 900 \text{ N}$ ,  
 $M_x = 600 \text{ N} \cdot \text{m}, M_y = -900 \text{ N} \cdot \text{m}$ ,  
 $M_z = 400 \text{ N} \cdot \text{m}$   
**7-45.** a.  $0 \leq x < a: V = \left(1 - \frac{a}{L}\right)P$ ,  
 $M = \left(1 - \frac{a}{L}\right)Px$ ,  
 $a < x \leq L: V = -\left(\frac{a}{L}\right)P$ ,  
 $M = P\left(a - \frac{a}{L}x\right)$

- b.**  $0 \leq x < 2 \text{ m}$ :  $V = 6 \text{ kN}$ ,  $M = \{6x\} \text{ kN}\cdot\text{m}$   
 $2 \text{ m} < x \leq 6 \text{ m}$ :  $V = -3 \text{ kN}$ ,  
 $M = \{18 - 3x\} \text{ kN}\cdot\text{m}$
- 7-46. a.** For  $0 \leq x < a$ ,  $V = P$ ,  $M = Px$ ,  
For  $a < x < L - a$ ,  $V = 0$ ,  $M = Pa$ ,  
For  $L - a < x \leq L$ ,  $V = -P$ ,  
 $M = P(L - x)$
- b.** For  $0 \leq x < 5 \text{ ft}$ ,  $V = 800 \text{ lb}$ ,  
 $M = 800x \text{ lb}\cdot\text{ft}$ ,  
For  $5 \text{ ft} < x < 7 \text{ ft}$ ,  $V = 0$ ,  
 $M = 4000 \text{ lb}\cdot\text{ft}$ ,  
For  $7 \text{ ft} < x \leq 12 \text{ ft}$ ,  $V = -800 \text{ lb}$ ,  
 $M = (9600 - 800x) \text{ lb}\cdot\text{ft}$
- 7-47. a.** For  $0 \leq x < a$ ,  $V = \frac{Pb}{a+b}$ ,  $M = \frac{Pb}{a+b}x$ ,  
For  $a < x \leq a+b$ ,  $V = -\frac{Pa}{a+b}$ ,  
 $M = Pa - \frac{Pa}{a+b}x$ ,
- b.** For  $0 \leq x < 5 \text{ ft}$ ,  $V = 350 \text{ lb}$ ,  
 $M = 350x \text{ lb}\cdot\text{ft}$ ,  
For  $5 \text{ ft} < x \leq 12 \text{ ft}$ ,  $V = -250 \text{ lb}$ ,  
 $M = 3000 - 250x \text{ lb}\cdot\text{ft}$
- 7-49.**  $0 \leq x < \frac{L}{3}$ :  $V = 0$ ,  $M = 0$ ,  
 $\frac{L}{3} < x < \frac{2L}{3}$ :  $V = 0$ ,  $M = M_0$ ,  
 $\frac{2L}{3} < x \leq L$ :  $V = 0$ ,  $M = 0$ ,  
 $0 \leq x < \frac{8}{3} \text{ m}$ :  $V = 0$ ,  $M = 0$ ,  
 $\frac{8}{3} \text{ m} < x < \frac{16}{3} \text{ m}$ :  $V = 0$ ,  $M = 500 \text{ N}\cdot\text{m}$ ,  
 $\frac{16}{3} \text{ m} < x \leq 8 \text{ m}$ :  $V = 0$ ,  $M = 0$
- 7-50.**  $M_{\max} = 2 \text{ kN}\cdot\text{m}$
- 7-51.**  $0 \leq x < a$ :  $V = -wx$ ,  $M = -\frac{w}{2}x^2$   
 $a < x \leq 2a$ :  $V = w(2a - x)$ ,  
 $M = 2wax - 2wa^2 - \frac{w}{2}x^2$
- 7-53.** For  $0 \leq x < 20 \text{ ft}$ ,  $V = \{490 - 50.0x\} \text{ lb}$ ,  
 $M = \{490x - 25.0x^2\} \text{ lb}\cdot\text{ft}$ ,  
For  $20 \text{ ft} < x \leq 30 \text{ ft}$ ,  $V = 0$ ,  $M = -200 \text{ lb}\cdot\text{ft}$
- 7-54. a.**  $V = \frac{w}{2}(L - 2x)$ ,  $M = \frac{w}{2}(Lx - x^2)$
- b.**  $V = (2500 - 500x) \text{ lb}$ ,  
 $M = (2500x - 250x^2) \text{ lb}\cdot\text{ft}$
- 7-55.** For  $0 \leq x < 8 \text{ m}$ ,  $V = (133.75 - 40x) \text{ kN}$ ,  
 $M = (133.75x - 20x^2) \text{ kN}\cdot\text{m}$ ,  
For  $8 \text{ m} < x \leq 11 \text{ m}$ ,  $V = 20 \text{ kN}$ ,  
 $M = (20x - 370) \text{ kN}\cdot\text{m}$
- 7-57.** For  $0 \leq x < L$ ,  $V = \frac{w}{18}(7L - 18x)$ ,  
 $M = \frac{w}{18}(7Lx - 9x^2)$ ,  
For  $L < x < 2L$ ,  
 $V = \frac{w}{2}(3L - 2x)$ ,  $M = \frac{w}{18}(27Lx - 20L^2 - 9x^2)$ ,  
For  $2L < x \leq 3L$ ,  $V = \frac{w}{18}(47L - 18x)$ ,  
 $M = \frac{w}{18}(47Lx - 9x^2 - 60L^2)$
- 7-58.** Member  $AB$ : For  $0 \leq x < 12 \text{ ft}$ ,  
 $V = \{875 - 150x\} \text{ lb}$ ,  
 $M = \{875x - 75.0x^2\} \text{ lb}\cdot\text{ft}$ ,  
For  $12 \text{ ft} < x \leq 14 \text{ ft}$ ,  $V = \{2100 - 150x\} \text{ lb}$ ,  
 $M = \{-75.0x^2 + 2100x - 14700\} \text{ lb}\cdot\text{ft}$ ,  
Member  $CBD$ : For  $0 \leq x < 2 \text{ ft}$ ,  $V = 919 \text{ lb}$ ,  
 $M = \{919x\} \text{ lb}\cdot\text{ft}$ , For  $2 \text{ ft} < x \leq 8 \text{ ft}$ ,  
 $V = -306 \text{ lb}$ ,  $M = \{2450 - 306x\} \text{ lb}\cdot\text{ft}$
- 7-59.** For  $0 \leq x < 9 \text{ ft}$ ,  $V = 25 - 1.67x^2$ ,  
 $M = 25x - 0.556x^3$
- 7-61.** For  $9 \text{ ft} < x \leq 13.5 \text{ ft}$ ,  $V = 0$ ,  $M = -180$
- 7-62.**  $x = 15^-$ ,  $V = -20$ ,  $M = -300$ ,  
 $x = 30^+$ ,  $V = 0$ ,  $M = 150$ ,  
 $x = 45^-$ ,  $V = -60$ ,  $M = -300$   
 $x = \frac{L}{2}$ ,  $P = \frac{4M_{\max}}{L}$
- 7-63.**  $0 \leq x \leq 12 \text{ ft}$ :  $V = \left\{ 48.0 - \frac{x^2}{6} \right\} \text{ kip}$ ,  
 $M = \left\{ 48.0x - \frac{x^3}{18} - 576 \right\} \text{ kip}\cdot\text{ft}$ ,  
 $12 < x \leq 24 \text{ ft}$ :  $V = \left\{ \frac{1}{6}(24 - x)^2 \right\} \text{ kip}$ ,  
 $M = \left\{ -\frac{1}{18}(24 - x)^3 \right\} \text{ kip}\cdot\text{ft}$
- 7-65.** For  $0 \leq x < 3 \text{ m}$ ,  $V = \{21.0 - 2x^2\} \text{ kN}$ ,  
 $M = \left\{ 21.0x - \frac{2}{3}x^3 \right\} \text{ kN}\cdot\text{m}$ ,  
For  $3 \text{ m} < x \leq 6 \text{ m}$ ,  $V = \{39.0 - 12x\} \text{ kN}$ ,  
 $M = \{-6x^2 + 39x - 18\} \text{ kN}\cdot\text{m}$
- 7-66.**  $V = \frac{w}{12L}(4L^2 - 6Lx - 3x^2)$ ,  
 $M = \frac{w}{12L}(4L^2x - 3Lx^2 - x^3)$ ,  $M_{\max} = 0.0940 wL^2$
- 7-67.**  $N = P \sin(\theta + \phi)$ ,  $V = -P \cos(\theta + \phi)$ ,  
 $M = Pr[\sin(\theta + \phi) - \sin \phi]$
- 7-69.**  $V_x = 0$ ,  $V_z = \{24.0 - 4y\} \text{ lb}$ ,  
 $M_x = \{2y^2 - 24y + 64.0\} \text{ lb}\cdot\text{ft}$ ,  
 $M_y = 8.00 \text{ lb}\cdot\text{ft}$ ,  $M_z = 0$
- 7-70.**  $x = 1^-$ ,  $V = 450 \text{ N}$ ,  $M = 450 \text{ N}\cdot\text{m}$ ,  
 $x = 3^+$ ,  $V = -950 \text{ N}$ ,  $M = 950 \text{ N}\cdot\text{m}$

- 7-71.**  $x = 1^-$ ,  $V = 600 \text{ N}$ ,  $M = 600 \text{ N} \cdot \text{m}$
- 7-74.**  $x = 0.5^+$ ,  $V = 450 \text{ N}$ ,  $M = -150 \text{ N} \cdot \text{m}$ ,  
 $x = 1.5^-$ ,  $V = -750 \text{ N}$ ,  $M = -300 \text{ N} \cdot \text{m}$
- 7-75.**  $x = 2^+$ ,  $V = -375 \text{ N}$ ,  $M = 750 \text{ N} \cdot \text{m}$
- 7-77.**  $x = 10^+$ ,  $V = 20.0 \text{ kip}$ ,  $M = -50.0 \text{ kip} \cdot \text{ft}$
- 7-78.**  $x = 2^+$ ,  $V = -14.3$ ,  $M = -8.6$
- 7-79.**  $x = 1^+$ ,  $V = 175$ ,  $M = -200$ ,  
 $x = 5^-$ ,  $V = -225$ ,  $M = -300$
- 7-81.**  $x = 4.5^-$ ,  $V = -31.5 \text{ kN}$ ,  $M = -45.0 \text{ kN} \cdot \text{m}$ ,  
 $x = 8.5^+$ ,  $V = 36.0 \text{ kN}$ ,  $M = -54.0 \text{ kN} \cdot \text{m}$
- 7-82.**  $x = 2.75$ ,  $V = 0$ ,  $M = 1356 \text{ N} \cdot \text{m}$
- 7-83.**  $x = 3$ ,  $V = -2.25 \text{ kN}$ ,  $M = 20.25 \text{ kN} \cdot \text{m}$
- 7-85.**  $x = 3^+$ ,  $V = 1800 \text{ lb}$ ,  $M = -900 \text{ lb} \cdot \text{ft}$   
 $x = 6$ ,  $V = 0$ ,  $M = 1800 \text{ lb} \cdot \text{ft}$
- 7-86.**  $x = 1.5$ ,  $V = 2.25 \text{ kN}$ ,  $M = -2.25 \text{ kN} \cdot \text{m}$
- 7-87.**  $x = 3$ ,  $V = 3.00 \text{ kN}$ ,  $M = -1.50 \text{ kN} \cdot \text{m}$
- 7-89.**  $x = \sqrt{15}$ ,  $V = 0$ ,  $M = 1291 \text{ lb} \cdot \text{ft}$   
 $x = 12^-$ ,  $V = -1900 \text{ lb}$ ,  $M = -6000 \text{ lb} \cdot \text{ft}$
- 7-90.**  $x = 0$ ,  $V = 13.5 \text{ kN}$ ,  $M = -9.5 \text{ kN} \cdot \text{m}$
- 7-91.**  $x = 3$ ,  $V = 0$ ,  $M = 18.0 \text{ kN} \cdot \text{m}$   
 $x = 6^-$ ,  $V = -27.0 \text{ kN}$ ,  $M = -18.0 \text{ kN} \cdot \text{m}$
- 7-93.**  $x = 15$ ,  $V = 0$ ,  $M = 37.5 \text{ kip} \cdot \text{ft}$
- 7-94.**  $y_B = 2.22 \text{ m}$ ,  $y_D = 1.55 \text{ m}$
- 7-95.**  $P_1 = 320 \text{ N}$ ,  $y_D = 2.33 \text{ m}$
- 7-97.**  $x_B = 5.39 \text{ m}$
- 7-98.**  $P = 700 \text{ N}$
- 7-99.**  $y_B = 8.67 \text{ ft}$ ,  $y_D = 7.04 \text{ ft}$
- 7-101.**  $y_B = 3.53 \text{ m}$ ,  $P = 0.8 \text{ kN}$ ,  $T_{\max} = T_{DE} = 8.17 \text{ kN}$
- 7-102.**  $w = 51.9 \text{ lb}/\text{ft}$
- 7-103.**  $T_{\max} = 14.4 \text{ kip}$ ,  $T_{\min} = 13.0 \text{ kip}$
- 7-105.**  $T_{AB} = T_{CD} = 212 \text{ lb}$  (max),  $y_B = 2 \text{ ft}$
- 7-106.**  $x = 2.57 \text{ ft}$ ,  $W = 247 \text{ lb}$
- 7-107.**  $T_A = 61.7 \text{ kip}$ ,  $T_B = 36.5 \text{ kip}$ ,  $T_C = 50.7 \text{ kip}$
- 7-109.**  $T_{\max} = 594 \text{ kN}$
- 7-110.**  $T_{\min} = 552 \text{ kN}$
- 7-111.**  $T_{\max} = 3.63 \text{ kip}$
- 7-113.**  $y = \frac{x^2}{7813} \left( 75 - \frac{x^2}{200} \right)$ ,  $T_{\max} = 9.28 \text{ kip}$
- 7-114.**  $h = 4.44 \text{ ft}$
- 7-115.**  $(F_h)_R = 6.25 \text{ kip}$ ,  $(F_v)_R = 2.51 \text{ kip}$
- 7-117.**  $(F_v)_A = 165 \text{ N}$ ,  $(F_h)_A = 73.9 \text{ N}$
- 7-118.**  $W = 4.00 \text{ kip}$ ,  $T_{\max} = 2.01 \text{ kip}$
- 7-121.**  $l = 104 \text{ ft}$
- 7-122.**  $h = 146 \text{ ft}$
- 7-123.**  $L = 302 \text{ ft}$

## Chapter 8

- 8-1.**  $P = 12.8 \text{ kN}$
- 8-2.**  $N_B = 2.43 \text{ kip}$ ,  $N_C = 1.62 \text{ kip}$ ,  $F = 200 \text{ lb}$
- 8-3.**  $N_A = 16.5 \text{ kN}$ ,  $N_B = 42.3 \text{ kN}$ ,  
It does not move.
- 8-5.**  $F = 2.76 \text{ kN}$

- 8-6.**  $F = 5.79 \text{ kN}$
- 8-7.** **a.** No  
**b.** Yes
- 8-10.**  $\phi = \theta$ ,  $P = W \sin(\alpha + \theta)$
- 8-11.** **a.**  $W = 318 \text{ lb}$   
**b.**  $W = 360 \text{ lb}$
- 8-13.**  $F_{CD} = 3.05 \text{ kN}$
- 8-14.**  $\theta = 21.8^\circ$
- 8-15.**  $l = 26.7 \text{ ft}$
- 8-17.**  $\mu_s = 0.231$
- 8-18.**  $P = 1350 \text{ lb}$
- 8-19.**  $N_A = 200 \text{ lb}$
- 8-21.**  $n = 12$
- 8-22.**  $\mu_s = 0.595$
- 8-23.**  $\theta = 33.4^\circ$
- 8-25.**  $d = 13.4 \text{ ft}$
- 8-26.**  $P = 740 \text{ N}$
- 8-27.**  $P = 860 \text{ N}$
- 8-29.**  $\theta = 11.0^\circ$
- 8-30.**  $\theta = 10.6^\circ$ ,  $x = 0.184 \text{ ft}$
- 8-31.**  $\theta = 8.53^\circ$ ,  $F_A = 1.48 \text{ lb}$ ,  $F_B = 0.890 \text{ lb}$
- 8-33.** No
- 8-34.** If  $P = \frac{1}{2}W$ ,  $\mu_s = \frac{1}{3}$   
If  $P \neq \frac{1}{2}W$ ,  

$$\mu_s = \frac{(P + W) - \sqrt{(W + 7P)(W - P)}}{2(2P - W)}$$
 for  $0 < P < W$
- 8-35.**  $P = 8.18 \text{ lb}$
- 8-37.**  $O_y = 400 \text{ N}$ ,  $O_x = 46.4 \text{ N}$
- 8-38.**  $P = 350 \text{ N}$ ,  $O_y = 945 \text{ N}$ ,  $O_x = 280 \text{ N}$
- 8-39.**  $\mu_s = 0.230$
- 8-41.**  $\theta = 31.0^\circ$
- 8-42.**  $P = 654 \text{ N}$
- 8-43.** The block fails to be in equilibrium.
- 8-45.**  $P = 355 \text{ N}$
- 8-46.**  $\mu_C = 0.0734$ ,  $\mu_B = 0.0964$
- 8-47.**  $\theta = 16.3^\circ$
- 8-49.** Yes
- 8-50.**  $m = 66.7 \text{ kg}$
- 8-51.**  $P = 408 \text{ N}$
- 8-53.**  $M = 55.2 \text{ lb} \cdot \text{ft}$
- 8-54.**  $\theta = 33.4^\circ$
- 8-55.**  $P = 13.3 \text{ lb}$
- 8-57.**  $P = 100 \text{ N}$ ,  $d = 1.50 \text{ ft}$
- 8-58.**  $\theta = 33.4^\circ$
- 8-59.**  $P = 5.53 \text{ kN}$ , yes
- 8-61.**  $P = 39.6 \text{ lb}$
- 8-62.**  $x = 18.3 \text{ mm}$
- 8-63.**  $P = 2.39 \text{ kN}$

- 8–65.**  $P = 4.05 \text{ kip}$   
**8–66.**  $P = 106 \text{ lb}$   
**8–67.**  $F = 66.7 \text{ N}$   
**8–69.**  $W = 7.19 \text{ kN}$   
**8–70.** The screw is self-locking.  
**8–71.**  $P = 617 \text{ lb}$   
**8–74.**  $M = 40.6 \text{ N} \cdot \text{m}$   
**8–75.**  $M = 48.3 \text{ N} \cdot \text{m}$   
**8–77.**  $\mu_s = 0.0637$   
**8–78.**  $M = 5.69 \text{ lb} \cdot \text{in.}$   
**8–79.**  $F = 1.98 \text{ kN}$   
**8–81.**  $F = 11.6 \text{ kN}$   
**8–82.**  $P = 104 \text{ N}$   
**8–83.** **a.**  $F = 1.31 \text{ kN}$   
**b.**  $F = 372 \text{ N}$   
**8–85.** He will successfully restrain the cow.  
**8–86.** Yes, it is possible.  
 $F = 137 \text{ lb}$   
**8–87.**  $T_1 = 57.7 \text{ lb}$   
**8–89.**  $m_A = 2.22 \text{ kg}$   
**8–90.**  $\theta = 99.2^\circ$   
**8–91.**  $n = 3 \text{ half turns}, N_m = 6.74 \text{ lb}$   
**8–93.**  $M = 458 \text{ N} \cdot \text{m}$   
**8–94.**  $W = 9.17 \text{ lb}$   
**8–95.**  $P = 78.7 \text{ lb}$   
**8–97.**  $M = 75.4 \text{ N} \cdot \text{m}, V = 0.171 \text{ m}^3$   
**8–99.**  $P = 53.6 \text{ N}$   
**8–101.**  $x = 0.384 \text{ m}$   
**8–102.**  $F_s = 85.4 \text{ N}$   
**8–103.**  $W_D = 12.7 \text{ lb}$   
**8–105.**  $\theta_{\max} = 38.8^\circ$   
**8–106.**  $M = 50.0 \text{ N} \cdot \text{m}, x = 286 \text{ mm}$   
**8–107.**  $M = 132 \text{ N} \cdot \text{m}$   
**8–109.**  $F = 10.7 \text{ lb}$   
**8–110.**  $M = 16.1 \text{ N} \cdot \text{m}$   
**8–111.**  $M = 237 \text{ N} \cdot \text{m}$   
**8–113.**  $M = \frac{2\mu_s PR}{3 \cos \theta}$   
**8–114.**  $T = 905 \text{ lb} \cdot \text{in.}$   
**8–115.**  $P = 118 \text{ N}$   
**8–117.**  $P = 29.0 \text{ lb}$   
**8–118.**  $M = \frac{8}{15} \mu_s PR$   
**8–119.**  $F = 18.9 \text{ N}$   
**8–121.**  $P = 20.5 \text{ lb}$   
**8–122.**  $T = 289 \text{ lb}, N = 479 \text{ lb}, F = 101 \text{ lb}$   
**8–123.**  $\mu_s = 0.0407$   
**8–125.**  $r = 20.6 \text{ mm}$   
**8–126.**  $P = 42.2 \text{ lb}$   
**8–127.**  $\mu_s = 0.411$   
**8–129.**  $P = 1333 \text{ lb}$   
**8–130.**  $P = 25.3 \text{ lb}$

**Chapter 9**

- 9–1.**  $\bar{x} = 124 \text{ mm}, \bar{y} = 0$   
**9–2.**  $\bar{x} = 0, \bar{y} = 1.82 \text{ ft}$   
**9–3.**  $\bar{x} = 0.574 \text{ m}, B_x = 0, A_y = 63.1 \text{ N}, B_y = 84.8 \text{ N}$   
**9–5.**  $\bar{y} = 0.857 \text{ m}$   
**9–6.**  $\bar{y} = \frac{2}{5} \text{ m}$   
**9–7.**  $\bar{x} = \frac{3}{8}a$   
**9–9.**  $\bar{x} = \frac{3}{2} \text{ m}$   
**9–10.**  $\bar{y} = \frac{12}{5} \text{ m}$   
**9–11.**  $\bar{x} = \frac{3}{4}b$   
**9–13.**  $\bar{x} = 6 \text{ m}$   
**9–14.**  $\bar{y} = 2.8 \text{ m}$   
**9–15.**  $\bar{x} = 0.398 \text{ m}$   
**9–17.**  $\bar{y} = 1.43 \text{ in.}$   
**9–18.**  $\bar{x} = \frac{a(1+n)}{2(2+n)}$   
**9–19.**  $\bar{y} = \frac{hn}{2n+1}$   
**9–21.**  $\bar{x} = 1\frac{3}{5} \text{ ft}$   
**9–22.**  $\bar{y} = 4\frac{8}{55} \text{ ft}$   
**9–23.**  $\bar{x} = \frac{3}{8}a$   
**9–25.**  $\bar{x} = 3.20 \text{ ft}, \bar{y} = 3.20 \text{ ft}, T_A = 384 \text{ lb}, T_C = 384 \text{ lb}, T_B = 1.15 \text{ kip}$   
**9–26.**  $\bar{x} = 3 \text{ ft}$   
**9–27.**  $\bar{y} = \frac{6}{5} \text{ ft}$   
**9–29.**  $\bar{y} = 40.0 \text{ mm}$   
**9–30.**  $\bar{x} = \frac{1}{3}(a+b)$   
**9–31.**  $\bar{y} = \frac{h}{3}$   
**9–33.**  $\bar{y} = \frac{\pi a}{8}$   
**9–34.**  $\bar{x} = 1.26 \text{ m}, \bar{y} = 0.143 \text{ m}, N_B = 47.9 \text{ kN}, A_x = 33.9 \text{ kN}, A_y = 73.9 \text{ kN}$   
**9–35.**  $\bar{x} = \left[ \frac{2(n+1)}{3(n+2)} \right] a$   
**9–37.**  $\bar{x} = \frac{2}{3} \left( \frac{r \sin \alpha}{\alpha} \right)$   
**9–38.**  $\bar{x} = 0.785 a$   
**9–39.**  $\bar{x} = \bar{y} = 0, \bar{z} = \frac{4}{3} \text{ m}$

**9-41.**  $\bar{z} = \frac{R^2 + 3r^2 + 2rR}{4(R^2 + r^2 + rR)}h$

**9-42.**  $\bar{y} = 2.61 \text{ ft}$

**9-43.**  $\bar{z} = \frac{h}{4}, \bar{x} = \bar{y} = \frac{a}{\pi}$

**9-45.**  $\bar{z} = \frac{4}{3} \text{ m}$

**9-46.**  $\bar{y} = \frac{3}{8}b, \bar{x} = \bar{z} = 0$

**9-47.**  $\bar{z} = 12.8 \text{ in.}$

**9-49.**  $\bar{z} = 0.675a$

**9-50.**  $\bar{z} = \frac{c}{4}$

**9-51.**  $d = 3 \text{ m}$

**9-53.**  $\bar{x} = 24.4 \text{ mm}, \bar{y} = 40.6 \text{ mm}$

**9-54.**  $\bar{x} = 0, \bar{y} = 58.3 \text{ mm}$

**9-55.**  $\bar{x} = 112 \text{ mm}, \bar{y} = 112 \text{ mm}, \bar{z} = 136 \text{ mm}$

**9-57.**  $\bar{x} = 0.200 \text{ m}, \bar{y} = 4.37 \text{ m}$

**9-58.**  $\bar{y} = 154 \text{ mm}$

**9-59.**  $\bar{x} = 0.571 \text{ in.}, \bar{y} = -0.571 \text{ in.}$

**9-61.**  $\bar{y} = 79.7 \text{ mm}$

**9-62.**  $\bar{x} = -1.00 \text{ in.}, \bar{y} = 4.625 \text{ in.}$

**9-63.**  $\bar{y} = 85.9 \text{ mm}$

**9-65.**  $\bar{x} = 1.57 \text{ in.}, \bar{y} = 1.57 \text{ in.}$

**9-66.**  $\bar{y} = 2 \text{ in.}$

**9-67.**  $\bar{y} = 272 \text{ mm}$

**9-69.**  $\bar{z} = 1.625 \text{ in.}$

**9-70.**  $\bar{z} = 4.32 \text{ in.}$

**9-71.**  $\bar{x} = \frac{\frac{2}{3}r \sin^3 \alpha}{\alpha - \frac{\sin 2\alpha}{2}}$

**9-73.**  $\bar{y} = \frac{\sqrt{2}(a^2 + at - t^2)}{2(2a - t)}$

**9-74.**  $\bar{x} = 2.81 \text{ ft}, \bar{y} = 1.73 \text{ ft}, N_B = 72.1 \text{ lb}, N_A = 86.9 \text{ lb}$

**9-75.**  $\bar{x} = 120 \text{ mm}, \bar{y} = 305 \text{ mm}, \bar{z} = 73.4 \text{ mm}$

**9-77.**  $\theta = 53.1^\circ$

**9-78.**  $\bar{z} = 2.48 \text{ ft}, \theta = 38.9^\circ$

**9-79.**  $\bar{z} = 0.70 \text{ ft}$

**9-81.**  $\bar{z} = 122 \text{ mm}$

**9-82.**  $h = 385 \text{ mm}$

**9-83.**  $\bar{x} = 5.07 \text{ ft}, \bar{y} = 3.80 \text{ ft}$

**9-85.**  $\bar{z} = 128 \text{ mm}$

**9-86.**  $\bar{z} = 754 \text{ mm}$

**9-87.**  $\bar{x} = 19.0 \text{ ft}, \bar{y} = 11.0 \text{ ft}$

**9-89.**  $\Sigma m = 16.4 \text{ kg}, \bar{x} = 153 \text{ mm}, \bar{y} = -15 \text{ mm}, \bar{z} = 111 \text{ mm}$

**9-90.**  $V = 27.2(10^3) \text{ ft}^3$

**9-91.**  $A = 3.56(10^3) \text{ ft}^2$

**9-93.**  $A = 4856 \text{ ft}^2$

**9-94.**  $W = 3.12(10^6) \text{ lb}$

**9-95.**  $V = \frac{\pi(6\pi + 4)}{6}a^3$

**9-97.**  $V = 0.114 \text{ m}^3$

**9-98.**  $A = 2.25 \text{ m}^2$

**9-99.**  $A = 276(10^3) \text{ mm}^2$

**9-101.**  $W = 84.7 \text{ kip}$

**9-102.** Number of gal. = 2.75 gal

**9-103.**  $A = 8\pi ba, V = 2\pi ba^2$

**9-105.**  $Q = 205 \text{ MJ}$

**9-106.**  $A = 119(10^3) \text{ mm}^2$

**9-107.**  $W = 126 \text{ kip}$

**9-109.**  $A = 1365 \text{ m}^2$

**9-110.**  $m = 138 \text{ kg}$

**9-111.**  $m = 2.68 \text{ kg}$

**9-113.**  $V = 1.40(10^3) \text{ in}^3$

**9-114.**  $h = 29.9 \text{ mm}$

**9-115.**  $F_R = 1250 \text{ lb}, \bar{x} = 2.33 \text{ ft}, \bar{y} = 4.33 \text{ ft}$

**9-117.**  $F_R = 24.0 \text{ kN},$

$\bar{x} = 2.00 \text{ m}, \bar{y} = 1.33 \text{ m}$

**9-118.**  $F_R = \frac{4ab}{\pi^2 p_0}, \bar{x} = \frac{a}{2}, \bar{y} = \frac{b}{2}$

**9-119.**  $F_{Rx} = 2rlp_0\left(\frac{\pi}{2}\right), F_R = \pi lrp_0$

**9-121.** For water:  $F_{R_A} = 157 \text{ kN}, F_{R_B} = 235 \text{ kN}$

For oil:  $d = 4.22 \text{ m}$

**9-122.**  $d = 2.61 \text{ m}$

**9-123.** F.S. = 2.71

**9-125.**  $F_1 = 9.60 \text{ kip}, F_2 = 40.3 \text{ kip}$

**9-126.**  $F_R = 427 \text{ lb}, \bar{y} = 1.71 \text{ ft}, \bar{x} = 0$

**9-127.**  $F_B = 29.4 \text{ kN}, F_A = 235 \text{ kN}$

**9-129.**  $F = 102 \text{ kN}$

**9-130.**  $F_{R_v} = 196 \text{ lb}, F_{R_h} = 125 \text{ lb}$

## Chapter 10

**10-1.**  $I_x = \frac{ab^3}{3(3n + 1)}$

**10-2.**  $I_y = \frac{a^3 b}{n + 3}$

**10-3.**  $I_x = 457(10^6) \text{ mm}^4$

**10-5.**  $I_x = 0.133 \text{ m}^4$

**10-6.**  $I_y = 0.286 \text{ m}^4$

**10-7.**  $I_x = 0.267 \text{ m}^4$

**10-9.**  $I_x = 23.8 \text{ ft}^4$

**10-10.**  $I_x = \frac{2}{15}bh^3$

**10-11.**  $I_x = 614 \text{ m}^4$

**10-13.**  $I_x = \frac{\pi}{8}m^4$

**10-14.**  $I_y = \frac{\pi}{2}m^4$

**10–15.**  $I_x = 205 \text{ in}^4$

**10–17.**  $I_x = \frac{1}{30}bh^3$

**10–18.**  $I_y = \frac{b^3h}{6}$

**10–19.**  $I_x = 0.267 \text{ m}^4$

**10–21.**  $I_x = 0.8 \text{ m}^4$

**10–22.**  $I_y = 0.571 \text{ m}^4$

**10–23.**  $I_x = \frac{3ab^3}{35}$

**10–25.**  $I_x = 209 \text{ in}^4$

**10–26.**  $I_y = 533 \text{ in}^4$

**10–27.**  $A = 14.0(10^3) \text{ mm}^2$

**10–29.**  $\bar{y} = 52.5 \text{ mm}, I_{x'} = 16.6(10^6) \text{ mm}^4, I_{y'} = 5.725(10^6) \text{ mm}^4$

**10–30.**  $I_x = 182 \text{ in}^4$

**10–31.**  $I_y = 966 \text{ in}^4$

**10–33.**  $\bar{I}_y = 2.03(10^9) \text{ mm}^4$

**10–34.**  $\bar{I}_y = 115(10^6) \text{ mm}^4$

**10–35.**  $\bar{y} = 207 \text{ mm}, \bar{I}_{x'} = 222(10^6) \text{ mm}^4$

**10–37.**  $I_y = 90.2(10^6) \text{ mm}^4$

**10–38.**  $I_x = 1971 \text{ in}^4$

**10–39.**  $I_y = 2376 \text{ in}^4$

**10–41.**  $I_y = 341 \text{ in}^4$

**10–42.**  $I_x = 154(10^6) \text{ mm}^4$

**10–43.**  $I_y = 91.3(10^6) \text{ mm}^4$

**10–45.**  $\bar{x} = 61.6 \text{ mm}, \bar{I}_{y'} = 41.2(10^6) \text{ mm}^4$

**10–46.**  $I_x = 1845 \text{ in}^4$

**10–47.**  $I_y = 522 \text{ in}^4$

**10–49.**  $I_{y'} = \frac{ab \sin \theta}{12}(b^2 + a^2 \cos^2 \theta)$

**10–50.**  $\bar{y} = 0.181 \text{ m}, \bar{I}_{x'} = 4.23(10^{-3}) \text{ m}^4$

**10–51.**  $\bar{I}_{x'} = 520(10^6) \text{ mm}^4$

**10–53.**  $I_y = 365 \text{ in}^4$

**10–54.**  $I_{xy} = \frac{1}{3}tl^3 \sin 2\theta$

**10–55.**  $I_{xy} = 5.06 \text{ in}^4$

**10–57.**  $I_{xy} = 10.7 \text{ m}^4, \bar{I}_{x'y'} = 1.07 \text{ m}^4$

**10–58.**  $I_{xy} = \frac{1}{6}a^2b^2$

**10–59.**  $I_{xy} = \frac{a^4}{280}$

**10–61.**  $I_{xy} = 0.667 \text{ in}^4$

**10–62.**  $I_{xy} = 97.8 \text{ in}^4$

**10–63.**  $I_u = 15.75 \text{ in}^4, I_u = 25.75 \text{ in}^4$

**10–65.**  $I_{xy} = 119 \text{ in}^4$

**10–66.**  $I_{xy} = 98.4(10^6) \text{ mm}^4$

**10–67.**  $\bar{x} = \bar{y} = 44.1 \text{ mm}, I_{x'y'} = -6.26(10^6) \text{ mm}^4$

**10–69.**  $I_u = 3.47(10^3) \text{ in}^4, I_v = 3.47(10^3) \text{ in}^4, I_{uv} = 2.05(10^3) \text{ in}^4$

**10–70.**  $I_u = 1.28(10^6) \text{ mm}^4, I_v = 3.31(10^6) \text{ mm}^4, I_{uv} = -1.75(10^6) \text{ mm}^4$

**10–71.**  $I_u = 1.28(10^6) \text{ mm}^4, I_{uv} = -1.75(10^6) \text{ mm}^4, I_v = 3.31(10^6) \text{ mm}^4$

**10–73.**  $I_{\max} = 1219 \text{ in}^4, I_{\min} = 36.3 \text{ in}^4, (\theta_p)_2 = 19.0^\circ \nearrow, (\theta_p)_1 = 71.0^\circ \nearrow$

**10–74.**  $I_{\max} = 17.4(10^6) \text{ mm}^4, I_{\min} = 1.84(10^6) \text{ mm}^4, (\theta_p)_1 = 60.0^\circ, (\theta_p)_2 = -30.0^\circ$

**10–75.**  $I_{\max} = 17.4(10^6) \text{ mm}^4, I_{\min} = 1.84(10^6) \text{ mm}^4, (\theta_p)_2 = 30.0^\circ \nearrow, (\theta_p)_1 = 60.0^\circ \nearrow$

**10–77.**  $I_{\max} = 250 \text{ in}^4, I_{\min} = 20.4 \text{ in}^4, (\theta_p)_2 = 22.5^\circ \nearrow, (\theta_p)_1 = 67.5^\circ \nearrow$

**10–78.**  $\theta = 6.08^\circ, I_{\max} = 1.74(10^3) \text{ in}^4, I_{\min} = 435 \text{ in}^4$

**10–79.**  $\theta = 6.08^\circ, I_{\max} = 1.74(10^3) \text{ in}^4, I_{\min} = 435 \text{ in}^4$

**10–81.**  $I_u = 11.8(10^6) \text{ mm}^4, I_{uv} = -5.09(10^6) \text{ mm}^4, I_v = 5.90(10^6) \text{ mm}^4$

**10–82.**  $\theta_{p1} = -31.4^\circ, \theta_{p2} = 58.6^\circ, I_{\max} = 309 \text{ in}^4, I_{\min} = 42.1 \text{ in}^4$

**10–83.**  $I_{\max} = 309 \text{ in}^4, I_{\min} = 42.1 \text{ in}^4, \theta_{p1} = -31.4^\circ, \theta_{p2} = 58.6^\circ$

**10–85.**  $I_x = \frac{2}{5}mb^2$

**10–86.**  $k_x = 57.7 \text{ mm}$

**10–87.**  $I_x = \frac{1}{3}ma^2$

**10–89.**  $I_x = \frac{2}{5}mb^2$

**10–90.**  $k_x = \sqrt{\frac{n+2}{2(n+4)}}h$

**10–91.**  $I_y = 2.25 \text{ slug} \cdot \text{ft}^2$

**10–93.**  $I_x = \frac{3}{10}mr^2$

**10–94.**  $I_y = 1.71(10^3) \text{ kg} \cdot \text{m}^2$

**10–95.**  $I_A = 0.0453 \text{ kg} \cdot \text{m}^2$

**10–97.**  $I_z = 1.53 \text{ kg} \cdot \text{m}^2$

**10–98.**  $\bar{y} = 1.78 \text{ m}, I_G = 4.45 \text{ kg} \cdot \text{m}^2$

**10–99.**  $I_O = 0.276 \text{ kg} \cdot \text{m}^2$

**10–101.**  $I_A = 222 \text{ slug} \cdot \text{ft}^2$

**10–102.**  $I_z = 29.4 \text{ kg} \cdot \text{m}^2$

**10–103.**  $I_O = \frac{1}{2}ma^2$

**10–105.**  $I_z = 0.113 \text{ kg} \cdot \text{m}^2$

**10–106.**  $I_G = 118 \text{ slug} \cdot \text{ft}^2$

**10–107.**  $I_O = 282 \text{ slug} \cdot \text{ft}^2$

**10–109.**  $I_z = 34.2 \text{ kg} \cdot \text{m}^2$

## Chapter 11

**11–1.**  $F_{AC} = 7.32 \text{ lb}$

**11–2.**  $F = 2P \cot \theta$

**11–3.**  $F_S = 15 \text{ lb}$

**11–5.**  $F = 369 \text{ N}$

**11-7.**  $M = 52.0 \text{ lb} \cdot \text{ft}$

**11-9.**  $\theta = 16.6^\circ, \theta = 35.8^\circ$

**11-10.**  $P = \frac{W}{2} \cot \theta$

**11-11.**  $\theta = 23.8^\circ, \theta = 72.3^\circ$

**11-13.**  $\theta = 90^\circ, \theta = 36.1^\circ$

**11-14.**  $k = 166 \text{ N/m}$

**11-15.**  $F = \frac{M}{2a \sin \theta}$

**11-17.**  $M = 13.1 \text{ N} \cdot \text{m}$

**11-18.**  $\theta = 41.2^\circ$

**11-19.**  $k = 9.88 \text{ kN/m}$

**11-21.** 
$$F = \frac{500\sqrt{0.04 \cos^2 \theta + 0.6}}{(0.2 \cos \theta + \sqrt{0.04 \cos^2 \theta + 0.6}) \sin \theta}$$

**11-22.**  $\theta = 9.21^\circ$

**11-23.**  $W_G = 2.5 \text{ lb}$

**11-25.** 
$$F = \frac{W(a + b - d \tan \theta)}{ac} \sqrt{a^2 + c^2 + 2ac \sin \theta}$$

**11-26.**  $x = -0.5 \text{ ft}$  unstable,  $x = 0.833 \text{ ft}$  stable

**11-27.** Unstable at  $\theta = 34.6^\circ$ , stable at  $\theta = 145^\circ$

**11-29.**  $\theta = 38.7^\circ$  unstable,  $\theta = 90^\circ$  stable,

$\theta = 141^\circ$  unstable

**11-30.**  $x = -0.424 \text{ ft}$  unstable,  $x = 0.590 \text{ ft}$  stable

**11-31.**  $\theta = 20.2^\circ$ , stable

**11-33.** Unstable equilibrium at  $\theta = 90^\circ$

Stable equilibrium at  $\theta = 49.0^\circ$

**11-34.** Unstable equilibrium at  $\theta = 0^\circ$

Stable equilibrium at  $\theta = 72.9^\circ$

**11-35.**  $k = 2.81 \text{ lb/ft}$

**11-37.** Stable equilibrium at  $\theta = 51.2^\circ$

Unstable equilibrium at  $\theta = 4.71^\circ$

**11-38.**  $k = 157 \text{ N/m}$

Stable equilibrium at  $\theta = 60^\circ$

**11-39.**  $W = \frac{8k}{3L}$

**11-41.** Stable equilibrium at  $\theta = 24.6^\circ$

**11-42.**  $\phi = 17.4^\circ, \theta = 9.18^\circ$

**11-43.** Unstable equilibrium at  $\theta = 23.2^\circ$

**11-45.**  $\theta = 0^\circ, \theta = 33.0^\circ$

**11-46.**  $m = 5.29 \text{ kg}$

**11-49.**  $\theta = 0^\circ, \theta = \cos^{-1}\left(\frac{d}{4a}\right)$

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ENGINEERING MECHANICS

# DYNAMICS

FOURTEENTH EDITION

# Chapter **12**



(© Lars Johansson/Fotolia)

Although each of these boats is rather large, from a distance their motion can be analyzed as if each were a particle.

# Kinematics of a Particle

## CHAPTER OBJECTIVES

- To introduce the concepts of position, displacement, velocity, and acceleration.
- To study particle motion along a straight line and represent this motion graphically.
- To investigate particle motion along a curved path using different coordinate systems.
- To present an analysis of dependent motion of two particles.
- To examine the principles of relative motion of two particles using translating axes.

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## 12.1 Introduction

**Mechanics** is a branch of the physical sciences that is concerned with the state of rest or motion of bodies subjected to the action of forces. Engineering mechanics is divided into two areas of study, namely, statics and dynamics. **Statics** is concerned with the equilibrium of a body that is either at rest or moves with constant velocity. Here we will consider **dynamics**, which deals with the accelerated motion of a body. The subject of dynamics will be presented in two parts: *kinematics*, which treats only the geometric aspects of the motion, and *kinetics*, which is the analysis of the forces causing the motion. To develop these principles, the dynamics of a particle will be discussed first, followed by topics in rigid-body dynamics in two and then three dimensions.

Historically, the principles of dynamics developed when it was possible to make an accurate measurement of time. Galileo Galilei (1564–1642) was one of the first major contributors to this field. His work consisted of experiments using pendulums and falling bodies. The most significant contributions in dynamics, however, were made by Isaac Newton (1642–1727), who is noted for his formulation of the three fundamental laws of motion and the law of universal gravitational attraction. Shortly after these laws were postulated, important techniques for their application were developed by Euler, D'Alembert, Lagrange, and others.

There are many problems in engineering whose solutions require application of the principles of dynamics. Typically the structural design of any vehicle, such as an automobile or airplane, requires consideration of the motion to which it is subjected. This is also true for many mechanical devices, such as motors, pumps, movable tools, industrial manipulators, and machinery. Furthermore, predictions of the motions of artificial satellites, projectiles, and spacecraft are based on the theory of dynamics. With further advances in technology, there will be an even greater need for knowing how to apply the principles of this subject.

**Problem Solving.** Dynamics is considered to be more involved than statics since both the forces applied to a body and its motion must be taken into account. Also, many applications require using calculus, rather than just algebra and trigonometry. In any case, the most effective way of learning the principles of dynamics is **to solve problems**. To be successful at this, it is necessary to present the work in a logical and orderly manner as suggested by the following sequence of steps:

1. Read the problem carefully and try to correlate the actual physical situation with the theory you have studied.
2. Draw any necessary diagrams and tabulate the problem data.
3. Establish a coordinate system and apply the relevant principles, generally in mathematical form.
4. Solve the necessary equations algebraically as far as practical; then, use a consistent set of units and complete the solution numerically. Report the answer with no more significant figures than the accuracy of the given data.
5. Study the answer using technical judgment and common sense to determine whether or not it seems reasonable.
6. Once the solution has been completed, review the problem. Try to think of other ways of obtaining the same solution.

In applying this general procedure, do the work as neatly as possible. Being neat generally stimulates clear and orderly thinking, and vice versa.

## 12.2 Rectilinear Kinematics: Continuous Motion

We will begin our study of dynamics by discussing the kinematics of a particle that moves along a rectilinear or straight-line path. Recall that a *particle* has a mass but negligible size and shape. Therefore we must limit application to those objects that have dimensions that are of no consequence in the analysis of the motion. In most problems, we will be interested in bodies of finite size, such as rockets, projectiles, or vehicles. Each of these objects can be considered as a particle, as long as the motion is characterized by the motion of its mass center and any rotation of the body is neglected.

**Rectilinear Kinematics.** The kinematics of a particle is characterized by specifying, at any given instant, the particle's position, velocity, and acceleration.

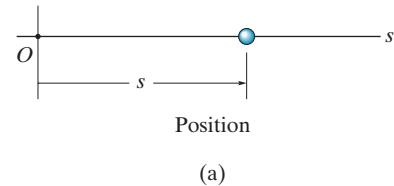
**Position.** The straight-line path of a particle will be defined using a single coordinate axis  $s$ , Fig. 12–1a. The origin  $O$  on the path is a fixed point, and from this point the **position coordinate**  $s$  is used to specify the location of the particle at any given instant. The magnitude of  $s$  is the distance from  $O$  to the particle, usually measured in meters (m) or feet (ft), and the sense of direction is defined by the algebraic sign on  $s$ . Although the choice is arbitrary, in this case  $s$  is positive since the coordinate axis is positive to the right of the origin. Likewise, it is negative if the particle is located to the left of  $O$ . Realize that *position is a vector quantity* since it has both magnitude and direction. Here, however, it is being represented by the algebraic scalar  $s$ , rather than in boldface  $\mathbf{s}$ , since the direction always remains along the coordinate axis.

**Displacement.** The **displacement** of the particle is defined as the *change in its position*. For example, if the particle moves from one point to another, Fig. 12–1b, the displacement is

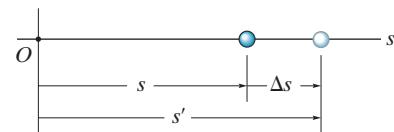
$$\Delta s = s' - s$$

In this case  $\Delta s$  is *positive* since the particle's final position is to the *right* of its initial position, i.e.,  $s' > s$ . Likewise, if the final position were to the *left* of its initial position,  $\Delta s$  would be *negative*.

The displacement of a particle is also a *vector quantity*, and it should be distinguished from the distance the particle travels. Specifically, the *distance traveled* is a *positive scalar* that represents the total length of path over which the particle travels.



(a)



Displacement

(b)

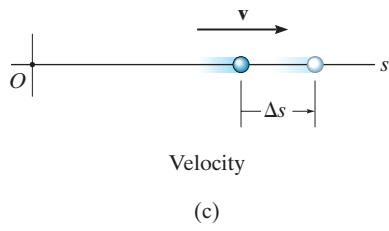
**Fig. 12–1**

**Velocity.** If the particle moves through a displacement  $\Delta s$  during the time interval  $\Delta t$ , the **average velocity** of the particle during this time interval is

$$v_{\text{avg}} = \frac{\Delta s}{\Delta t}$$

If we take smaller and smaller values of  $\Delta t$ , the magnitude of  $\Delta s$  becomes smaller and smaller. Consequently, the **instantaneous velocity** is a vector defined as  $v = \lim_{\Delta t \rightarrow 0} (\Delta s / \Delta t)$ , or

$$( \pm ) \quad v = \frac{ds}{dt} \quad (12-1)$$



Since  $\Delta t$  or  $dt$  is always positive, the sign used to define the *sense* of the velocity is the same as that of  $\Delta s$  or  $ds$ . For example, if the particle is moving to the *right*, Fig. 12-1c, the velocity is *positive*; whereas if it is moving to the *left*, the velocity is *negative*. (This is emphasized here by the arrow written at the left of Eq. 12-1.) The *magnitude* of the velocity is known as the **speed**, and it is generally expressed in units of m/s or ft/s.

Occasionally, the term “average speed” is used. The **average speed** is always a positive scalar and is defined as the total distance traveled by a particle,  $s_T$ , divided by the elapsed time  $\Delta t$ ; i.e.,

$$(v_{\text{sp}})_{\text{avg}} = \frac{s_T}{\Delta t}$$

For example, the particle in Fig. 12-1d travels along the path of length  $s_T$  in time  $\Delta t$ , so its average speed is  $(v_{\text{sp}})_{\text{avg}} = s_T / \Delta t$ , but its average velocity is  $v_{\text{avg}} = -\Delta s / \Delta t$ .

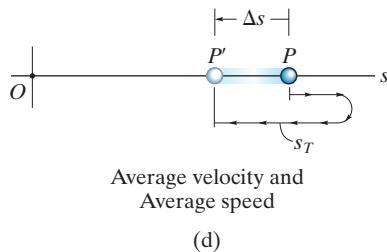


Fig. 12-1 (cont.)

**Acceleration.** Provided the velocity of the particle is known at two points, the *average acceleration* of the particle during the time interval  $\Delta t$  is defined as

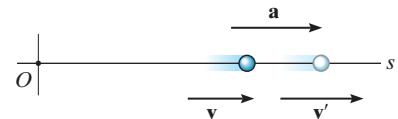
$$a_{\text{avg}} = \frac{\Delta v}{\Delta t}$$

Here  $\Delta v$  represents the difference in the velocity during the time interval  $\Delta t$ , i.e.,  $\Delta v = v' - v$ , Fig. 12-1e.

The *instantaneous acceleration* at time  $t$  is a vector that is found by taking smaller and smaller values of  $\Delta t$  and corresponding smaller and smaller values of  $\Delta v$ , so that  $a = \lim_{\Delta t \rightarrow 0} (\Delta v / \Delta t)$ , or

(±)

$$a = \frac{dv}{dt} \quad (12-2)$$



Acceleration

(e)

Substituting Eq. 12-1 into this result, we can also write

(±)

$$a = \frac{d^2s}{dt^2}$$

Both the average and instantaneous acceleration can be either positive or negative. In particular, when the particle is *slowing down*, or its speed is decreasing, the particle is said to be *decelerating*. In this case,  $v'$  in Fig. 12-1f is less than  $v$ , and so  $\Delta v = v' - v$  will be negative. Consequently,  $a$  will also be negative, and therefore it will act to the *left*, in the *opposite sense* to  $v$ . Also, notice that if the particle is originally at rest, then it can have an acceleration if a moment later it has a velocity  $v'$ ; and, if the *velocity is constant*, then the *acceleration is zero* since  $\Delta v = v - v = 0$ . Units commonly used to express the magnitude of acceleration are  $\text{m/s}^2$  or  $\text{ft/s}^2$ .

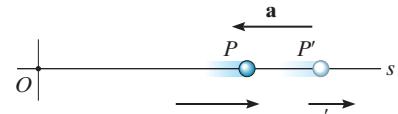
Finally, an important differential relation involving the displacement, velocity, and acceleration along the path may be obtained by eliminating the time differential  $dt$  between Eqs. 12-1 and 12-2. We have

$$dt = \frac{ds}{v} = \frac{dv}{a}$$

or

(±)

$$a ds = v dv \quad (12-3)$$



Deceleration

(f)

Fig. 12-1 (cont.)



When the ball is released, it has zero velocity but an acceleration of  $9.81 \text{ m/s}^2$ .  
 (© R.C. Hibbeler)

**Constant Acceleration,  $a = a_c$ .** When the acceleration is constant, each of the three kinematic equations  $a_c = dv/dt$ ,  $v = ds/dt$ , and  $a_c ds = v dv$  can be integrated to obtain formulas that relate  $a_c$ ,  $v$ ,  $s$ , and  $t$ .

**Velocity as a Function of Time.** Integrate  $a_c = dv/dt$ , assuming that initially  $v = v_0$  when  $t = 0$ .

$$\int_{v_0}^v dv = \int_0^t a_c dt$$

$$v = v_0 + a_c t \quad \text{Constant Acceleration} \quad (12-4)$$

**Position as a Function of Time.** Integrate  $v = ds/dt = v_0 + a_c t$ , assuming that initially  $s = s_0$  when  $t = 0$ .

$$\int_{s_0}^s ds = \int_0^t (v_0 + a_c t) dt$$

$$s = s_0 + v_0 t + \frac{1}{2} a_c t^2 \quad \text{Constant Acceleration} \quad (12-5)$$

**Velocity as a Function of Position.** Either solve for  $t$  in Eq. 12-4 and substitute into Eq. 12-5, or integrate  $v dv = a_c ds$ , assuming that initially  $v = v_0$  at  $s = s_0$ .

$$\int_{v_0}^v v dv = \int_{s_0}^s a_c ds$$

$$v^2 = v_0^2 + 2a_c(s - s_0) \quad \text{Constant Acceleration} \quad (12-6)$$

The algebraic signs of  $s_0$ ,  $v_0$ , and  $a_c$ , used in the above three equations, are determined from the positive direction of the  $s$  axis as indicated by the arrow written at the left of each equation. Remember that these equations are useful *only when the acceleration is constant and when  $t = 0, s = s_0, v = v_0$* . A typical example of constant accelerated motion occurs when a body falls freely toward the earth. If air resistance is neglected and the distance of fall is short, then the *downward* acceleration of the body when it is close to the earth is constant and approximately  $9.81 \text{ m/s}^2$  or  $32.2 \text{ ft/s}^2$ . The proof of this is given in Example 13.2.

## Important Points

- Dynamics is concerned with bodies that have accelerated motion.
- Kinematics is a study of the geometry of the motion.
- Kinetics is a study of the forces that cause the motion.
- Rectilinear kinematics refers to straight-line motion.
- Speed refers to the magnitude of velocity.
- Average speed is the total distance traveled divided by the total time. This is different from the average velocity, which is the displacement divided by the time.
- A particle that is slowing down is decelerating.
- A particle can have an acceleration and yet have zero velocity.
- The relationship  $a ds = v dv$  is derived from  $a = dv/dt$  and  $v = ds/dt$ , by eliminating  $dt$ .



During the time this rocket undergoes rectilinear motion, its altitude as a function of time can be measured and expressed as  $s = s(t)$ . Its velocity can then be found using  $v = ds/dt$ , and its acceleration can be determined from  $a = dv/dt$ .  
© NASA

## Procedure for Analysis

### Coordinate System.

- Establish a position coordinate  $s$  along the path and specify its *fixed origin* and positive direction.
- Since motion is along a straight line, the vector quantities position, velocity, and acceleration can be represented as algebraic scalars. For analytical work the sense of  $s$ ,  $v$ , and  $a$  is then defined by their *algebraic signs*.
- The positive sense for each of these scalars can be indicated by an arrow shown alongside each kinematic equation as it is applied.

### Kinematic Equations.

- If a relation is known between any *two* of the four variables  $a$ ,  $v$ ,  $s$ , and  $t$ , then a third variable can be obtained by using one of the kinematic equations,  $a = dv/dt$ ,  $v = ds/dt$  or  $a ds = v dv$ , since each equation relates all three variables.\*
- Whenever integration is performed, it is important that the position and velocity be known at a given instant in order to evaluate either the constant of integration if an indefinite integral is used, or the limits of integration if a definite integral is used.
- Remember that Eqs. 12–4 through 12–6 have only limited use. These equations apply *only* when the *acceleration is constant* and the initial conditions are  $s = s_0$  and  $v = v_0$  when  $t = 0$ .

\*Some standard differentiation and integration formulas are given in Appendix A.

## EXAMPLE 12.1



(© R.C. Hibbeler)

The car on the left in the photo and in Fig. 12–2 moves in a straight line such that for a short time its velocity is defined by  $v = (3t^2 + 2t)$  ft/s, where  $t$  is in seconds. Determine its position and acceleration when  $t = 3$  s. When  $t = 0$ ,  $s = 0$ .

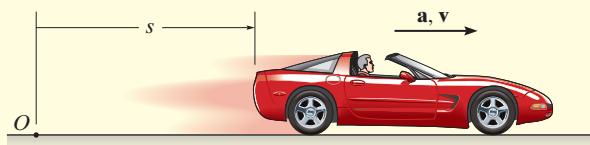


Fig. 12–2

## SOLUTION

**Coordinate System.** The position coordinate extends from the fixed origin  $O$  to the car, positive to the right.

**Position.** Since  $v = f(t)$ , the car's position can be determined from  $v = ds/dt$ , since this equation relates  $v$ ,  $s$ , and  $t$ . Noting that  $s = 0$  when  $t = 0$ , we have\*

$$\begin{aligned}
 (\pm) \qquad \qquad \qquad v &= \frac{ds}{dt} = (3t^2 + 2t) \\
 \int_0^s ds &= \int_0^t (3t^2 + 2t) dt \\
 s \Big|_0^s &= t^3 + t^2 \Big|_0^t \\
 s &= t^3 + t^2
 \end{aligned}$$

When  $t = 3$  s,

$$s = (3)^3 + (3)^2 = 36 \text{ ft} \qquad \text{Ans.}$$

**Acceleration.** Since  $v = f(t)$ , the acceleration is determined from  $a = dv/dt$ , since this equation relates  $a$ ,  $v$ , and  $t$ .

$$\begin{aligned}
 (\pm) \qquad \qquad \qquad a &= \frac{dv}{dt} = \frac{d}{dt}(3t^2 + 2t) \\
 &= 6t + 2
 \end{aligned}$$

When  $t = 3$  s,

$$a = 6(3) + 2 = 20 \text{ ft/s}^2 \rightarrow \qquad \text{Ans.}$$

**NOTE:** The formulas for constant acceleration *cannot* be used to solve this problem, because the acceleration is a function of time.

\*The *same result* can be obtained by evaluating a constant of integration  $C$  rather than using definite limits on the integral. For example, integrating  $ds = (3t^2 + 2t)dt$  yields  $s = t^3 + t^2 + C$ . Using the condition that at  $t = 0$ ,  $s = 0$ , then  $C = 0$ .

**EXAMPLE | 12.2**

A small projectile is fired vertically *downward* into a fluid medium with an initial velocity of 60 m/s. Due to the drag resistance of the fluid the projectile experiences a deceleration of  $a = (-0.4v^3)$  m/s<sup>2</sup>, where  $v$  is in m/s. Determine the projectile's velocity and position 4 s after it is fired.

**SOLUTION**

**Coordinate System.** Since the motion is downward, the position coordinate is positive downward, with origin located at  $O$ , Fig. 12-3.

**Velocity.** Here  $a = f(v)$  and so we must determine the velocity as a function of time using  $a = dv/dt$ , since this equation relates  $v$ ,  $a$ , and  $t$ . (Why not use  $v = v_0 + a_c t$ ?) Separating the variables and integrating, with  $v_0 = 60$  m/s when  $t = 0$ , yields

$$(+\downarrow) \quad a = \frac{dv}{dt} = -0.4v^3$$

$$\int_{60 \text{ m/s}}^v \frac{dv}{-0.4v^3} = \int_0^t dt$$

$$\frac{1}{-0.4} \left( \frac{1}{-2} \right) \frac{1}{v^2} \Big|_{60}^v = t - 0$$

$$\frac{1}{0.8} \left[ \frac{1}{v^2} - \frac{1}{(60)^2} \right] = t$$

$$v = \left\{ \left[ \frac{1}{(60)^2} + 0.8t \right]^{-1/2} \right\} \text{ m/s}$$

Here the positive root is taken, since the projectile will continue to move downward. When  $t = 4$  s,

$$v = 0.559 \text{ m/s} \downarrow \quad \text{Ans.}$$

**Position.** Knowing  $v = f(t)$ , we can obtain the projectile's position from  $v = ds/dt$ , since this equation relates  $s$ ,  $v$ , and  $t$ . Using the initial condition  $s = 0$ , when  $t = 0$ , we have

$$(+\downarrow) \quad v = \frac{ds}{dt} = \left[ \frac{1}{(60)^2} + 0.8t \right]^{-1/2}$$

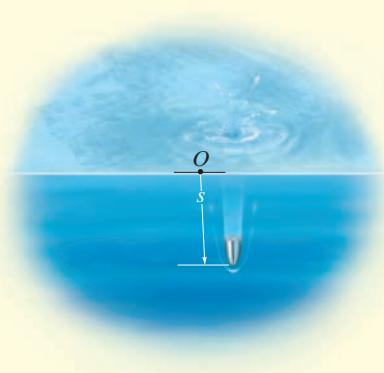
$$\int_0^s ds = \int_0^t \left[ \frac{1}{(60)^2} + 0.8t \right]^{-1/2} dt$$

$$s = \frac{2}{0.8} \left[ \frac{1}{(60)^2} + 0.8t \right]^{1/2} \Big|_0^t$$

$$s = \frac{1}{0.4} \left\{ \left[ \frac{1}{(60)^2} + 0.8t \right]^{1/2} - \frac{1}{60} \right\} \text{ m}$$

When  $t = 4$  s,

$$s = 4.43 \text{ m} \quad \text{Ans.}$$

**Fig. 12-3**

## EXAMPLE 12.3

During a test a rocket travels upward at 75 m/s, and when it is 40 m from the ground its engine fails. Determine the maximum height  $s_B$  reached by the rocket and its speed just before it hits the ground. While in motion the rocket is subjected to a constant downward acceleration of  $9.81 \text{ m/s}^2$  due to gravity. Neglect the effect of air resistance.

## SOLUTION

**Coordinate System.** The origin  $O$  for the position coordinate  $s$  is taken at ground level with positive upward, Fig. 12–4.

**Maximum Height.** Since the rocket is traveling *upward*,  $v_A = +75 \text{ m/s}$  when  $t = 0$ . At the maximum height  $s = s_B$  the velocity  $v_B = 0$ . For the entire motion, the acceleration is  $a_c = -9.81 \text{ m/s}^2$  (negative since it acts in the *opposite* sense to positive velocity or positive displacement). Since  $a_c$  is *constant* the rocket's position may be related to its velocity at the two points  $A$  and  $B$  on the path by using Eq. 12–6, namely,

$$\begin{aligned} (+\uparrow) \quad v_B^2 &= v_A^2 + 2a_c(s_B - s_A) \\ 0 &= (75 \text{ m/s})^2 + 2(-9.81 \text{ m/s}^2)(s_B - 40 \text{ m}) \\ s_B &= 327 \text{ m} \end{aligned} \quad \text{Ans.}$$

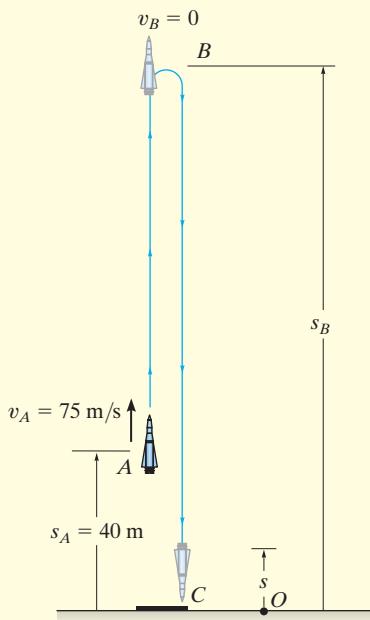


Fig. 12–4

**Velocity.** To obtain the velocity of the rocket just before it hits the ground, we can apply Eq. 12–6 between points  $B$  and  $C$ , Fig. 12–4.

$$\begin{aligned} (+\uparrow) \quad v_C^2 &= v_B^2 + 2a_c(s_C - s_B) \\ &= 0 + 2(-9.81 \text{ m/s}^2)(0 - 327 \text{ m}) \\ v_C &= -80.1 \text{ m/s} = 80.1 \text{ m/s} \downarrow \end{aligned} \quad \text{Ans.}$$

The negative root was chosen since the rocket is moving downward.

Similarly, Eq. 12–6 may also be applied between points  $A$  and  $C$ , i.e.,

$$\begin{aligned} (+\uparrow) \quad v_C^2 &= v_A^2 + 2a_c(s_C - s_A) \\ &= (75 \text{ m/s})^2 + 2(-9.81 \text{ m/s}^2)(0 - 40 \text{ m}) \\ v_C &= -80.1 \text{ m/s} = 80.1 \text{ m/s} \downarrow \end{aligned} \quad \text{Ans.}$$

**NOTE:** It should be realized that the rocket is subjected to a *deceleration* from  $A$  to  $B$  of  $9.81 \text{ m/s}^2$ , and then from  $B$  to  $C$  it is *accelerated* at this rate. Furthermore, even though the rocket momentarily comes to *rest* at  $B$  ( $v_B = 0$ ) the acceleration at  $B$  is still  $9.81 \text{ m/s}^2$  downward!

**EXAMPLE | 12.4**

A metallic particle is subjected to the influence of a magnetic field as it travels downward through a fluid that extends from plate *A* to plate *B*, Fig. 12–5. If the particle is released from rest at the midpoint *C*,  $s = 100$  mm, and the acceleration is  $a = (4s)$  m/s<sup>2</sup>, where  $s$  is in meters, determine the velocity of the particle when it reaches plate *B*,  $s = 200$  mm, and the time it takes to travel from *C* to *B*.

**SOLUTION**

**Coordinate System.** As shown in Fig. 12–5,  $s$  is positive downward, measured from plate *A*.

**Velocity.** Since  $a = f(s)$ , the velocity as a function of position can be obtained by using  $v dv = a ds$ . Realizing that  $v = 0$  at  $s = 0.1$  m, we have

$$\begin{aligned} (+\downarrow) \quad v dv &= a ds \\ \int_0^v v dv &= \int_{0.1 \text{ m}}^s 4s ds \\ \frac{1}{2} v^2 \Big|_0^v &= \frac{4}{2} s^2 \Big|_{0.1 \text{ m}}^s \\ v &= 2(s^2 - 0.01)^{1/2} \text{ m/s} \end{aligned} \quad (1)$$

At  $s = 200$  mm = 0.2 m,

$$v_B = 0.346 \text{ m/s} = 346 \text{ mm/s} \downarrow \quad \text{Ans.}$$

The positive root is chosen since the particle is traveling downward, i.e., in the  $+s$  direction.

**Time.** The time for the particle to travel from *C* to *B* can be obtained using  $v = ds/dt$  and Eq. 1, where  $s = 0.1$  m when  $t = 0$ . From Appendix A,

$$\begin{aligned} (+\downarrow) \quad ds &= v dt \\ &= 2(s^2 - 0.01)^{1/2} dt \\ \int_{0.1}^s \frac{ds}{(s^2 - 0.01)^{1/2}} &= \int_0^t 2 dt \\ \ln(\sqrt{s^2 - 0.01} + s) \Big|_{0.1}^s &= 2t \Big|_0^t \\ \ln(\sqrt{s^2 - 0.01} + s) + 2.303 &= 2t \end{aligned}$$

At  $s = 0.2$  m,

$$t = \frac{\ln(\sqrt{(0.2)^2 - 0.01} + 0.2) + 2.303}{2} = 0.658 \text{ s} \quad \text{Ans.}$$

**NOTE:** The formulas for constant acceleration cannot be used here because the acceleration changes with position, i.e.,  $a = 4s$ .

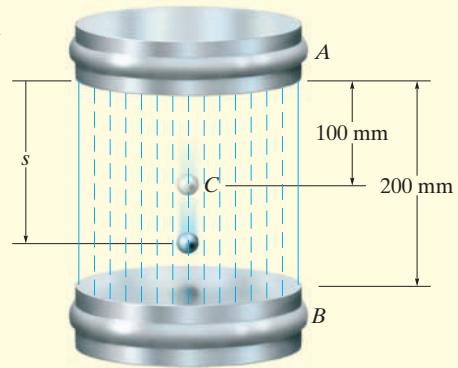


Fig. 12–5

## EXAMPLE 12.5

A particle moves along a horizontal path with a velocity of  $v = (3t^2 - 6t)$  m/s, where  $t$  is the time in seconds. If it is initially located at the origin  $O$ , determine the distance traveled in 3.5 s, and the particle's average velocity and average speed during the time interval.

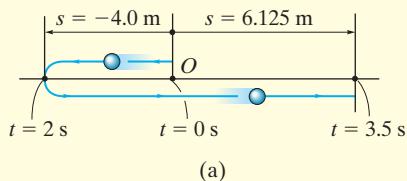
## SOLUTION

**Coordinate System.** Here positive motion is to the right, measured from the origin  $O$ , Fig. 12-6a.

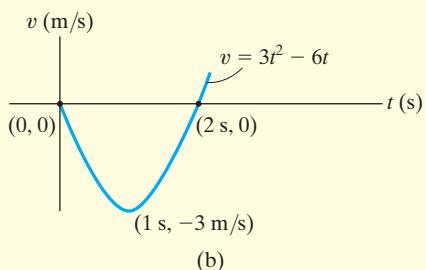
**Distance Traveled.** Since  $v = f(t)$ , the position as a function of time may be found by integrating  $v = ds/dt$  with  $t = 0, s = 0$ .

( $\Rightarrow$ )

$$\begin{aligned} ds &= v dt \\ &= (3t^2 - 6t) dt \\ \int_0^s ds &= \int_0^t (3t^2 - 6t) dt \\ s &= (t^3 - 3t^2) \text{ m} \end{aligned} \quad (1)$$



(a)



(b)

Fig. 12-6

In order to determine the distance traveled in 3.5 s, it is necessary to investigate the path of motion. If we consider a graph of the velocity function, Fig. 12-6b, then it reveals that for  $0 < t < 2$  s the velocity is *negative*, which means the particle is traveling to the *left*, and for  $t > 2$  s the velocity is *positive*, and hence the particle is traveling to the *right*. Also, note that  $v = 0$  at  $t = 2$  s. The particle's position when  $t = 0$ ,  $t = 2$  s, and  $t = 3.5$  s can be determined from Eq. 1. This yields

$$s|_{t=0} = 0 \quad s|_{t=2 \text{ s}} = -4.0 \text{ m} \quad s|_{t=3.5 \text{ s}} = 6.125 \text{ m}$$

The path is shown in Fig. 12-6a. Hence, the distance traveled in 3.5 s is

$$s_T = 4.0 + 4.0 + 6.125 = 14.125 \text{ m} = 14.1 \text{ m} \quad \text{Ans.}$$

**Velocity.** The *displacement* from  $t = 0$  to  $t = 3.5$  s is

$$\Delta s = s|_{t=3.5 \text{ s}} - s|_{t=0} = 6.125 \text{ m} - 0 = 6.125 \text{ m}$$

and so the average velocity is

$$v_{\text{avg}} = \frac{\Delta s}{\Delta t} = \frac{6.125 \text{ m}}{3.5 \text{ s} - 0} = 1.75 \text{ m/s} \rightarrow \quad \text{Ans.}$$

The average speed is defined in terms of the *distance traveled*  $s_T$ . This positive scalar is

$$(v_{\text{sp}})_{\text{avg}} = \frac{s_T}{\Delta t} = \frac{14.125 \text{ m}}{3.5 \text{ s} - 0} = 4.04 \text{ m/s} \quad \text{Ans.}$$

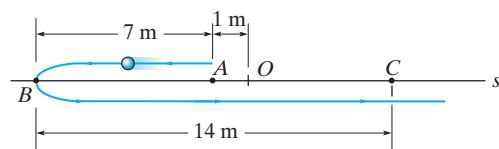
**NOTE:** In this problem, the acceleration is  $a = dv/dt = (6t - 6)$  m/s<sup>2</sup>, which is not constant.

It is highly suggested that you test yourself on the solutions to these examples, by covering them over and then trying to think about which equations of kinematics must be used and how they are applied in order to determine the unknowns. Then before solving any of the problems, try and solve some of the Preliminary and Fundamental Problems which follow. The solutions and answers to all these problems are given in the back of the book. **Doing this throughout the book will help immensely in understanding how to apply the theory, and thereby develop your problem-solving skills.**

## PRELIMINARY PROBLEM

### P12-1.

- a) If  $s = (2t^3)$  m, where  $t$  is in seconds, determine  $v$  when  $t = 2$  s.
- b) If  $v = (5s)$  m/s, where  $s$  is in meters, determine  $a$  at  $s = 1$  m.
- c) If  $v = (4t + 5)$  m/s, where  $t$  is in seconds, determine  $a$  when  $t = 2$  s.
- d) If  $a = 2 \text{ m/s}^2$ , determine  $v$  when  $t = 2$  s if  $v = 0$  when  $t = 0$ .
- e) If  $a = 2 \text{ m/s}^2$ , determine  $v$  at  $s = 4$  m if  $v = 3 \text{ m/s}$  at  $s = 0$ .
- f) If  $a = (s) \text{ m/s}^2$ , where  $s$  is in meters, determine  $v$  when  $s = 5$  m if  $v = 0$  at  $s = 4$  m.
- g) If  $a = 4 \text{ m/s}^2$ , determine  $s$  when  $t = 3$  s if  $v = 2 \text{ m/s}$  and  $s = 2 \text{ m}$  when  $t = 0$ .
- h) If  $a = (8t^2) \text{ m/s}^2$ , determine  $v$  when  $t = 1$  s if  $v = 0$  at  $t = 0$ .
- i) If  $s = (3t^2 + 2)$  m, determine  $v$  when  $t = 2$  s.
- j) When  $t = 0$  the particle is at  $A$ . In four seconds it travels to  $B$ , then in another six seconds it travels to  $C$ . Determine the average velocity and the average speed. The origin of the coordinate is at  $O$ .



**Prob. P12-1**

## FUNDAMENTAL PROBLEMS

**F12–1.** Initially, the car travels along a straight road with a speed of 35 m/s. If the brakes are applied and the speed of the car is reduced to 10 m/s in 15 s, determine the constant deceleration of the car.



Prob. F12–1

**F12–5.** The position of the particle is given by  $s = (2t^2 - 8t + 6)$  m, where  $t$  is in seconds. Determine the time when the velocity of the particle is zero, and the total distance traveled by the particle when  $t = 3$  s.



Prob. F12–5

**F12–2.** A ball is thrown vertically upward with a speed of 15 m/s. Determine the time of flight when it returns to its original position.



Prob. F12–2

**F12–6.** A particle travels along a straight line with an acceleration of  $a = (10 - 0.2s)$  m/s<sup>2</sup>, where  $s$  is measured in meters. Determine the velocity of the particle when  $s = 10$  m if  $v = 5$  m/s at  $s = 0$ .



Prob. F12–6

**F12–3.** A particle travels along a straight line with a velocity of  $v = (4t - 3t^2)$  m/s, where  $t$  is in seconds. Determine the position of the particle when  $t = 4$  s.  $s = 0$  when  $t = 0$ .

**F12–4.** A particle travels along a straight line with a speed  $v = (0.5t^3 - 8t)$  m/s, where  $t$  is in seconds. Determine the acceleration of the particle when  $t = 2$  s.

**F12–7.** A particle moves along a straight line such that its acceleration is  $a = (4t^2 - 2)$  m/s<sup>2</sup>, where  $t$  is in seconds. When  $t = 0$ , the particle is located 2 m to the left of the origin, and when  $t = 2$  s, it is 20 m to the left of the origin. Determine the position of the particle when  $t = 4$  s.

**F12–8.** A particle travels along a straight line with a velocity of  $v = (20 - 0.05s^2)$  m/s, where  $s$  is in meters. Determine the acceleration of the particle at  $s = 15$  m.

## PROBLEMS

**12–1.** Starting from rest, a particle moving in a straight line has an acceleration of  $a = (2t - 6)$  m/s<sup>2</sup>, where  $t$  is in seconds. What is the particle's velocity when  $t = 6$  s, and what is its position when  $t = 11$  s?

**12–2.** If a particle has an initial velocity of  $v_0 = 12$  ft/s to the right, at  $s_0 = 0$ , determine its position when  $t = 10$  s, if  $a = 2$  ft/s<sup>2</sup> to the left.

**12–3.** A particle travels along a straight line with a velocity  $v = (12 - 3t^2)$  m/s, where  $t$  is in seconds. When  $t = 1$  s, the particle is located 10 m to the left of the origin. Determine the acceleration when  $t = 4$  s, the displacement from  $t = 0$  to  $t = 10$  s, and the distance the particle travels during this time period.

**\*12–4.** A particle travels along a straight line with a constant acceleration. When  $s = 4$  ft,  $v = 3$  ft/s and when  $s = 10$  ft,  $v = 8$  ft/s. Determine the velocity as a function of position.

**12–5.** The velocity of a particle traveling in a straight line is given by  $v = (6t - 3t^2)$  m/s, where  $t$  is in seconds. If  $s = 0$  when  $t = 0$ , determine the particle's deceleration and position when  $t = 3$  s. How far has the particle traveled during the 3-s time interval, and what is its average speed?

**12–6.** The position of a particle along a straight line is given by  $s = (1.5t^3 - 13.5t^2 + 22.5t)$  ft, where  $t$  is in seconds. Determine the position of the particle when  $t = 6$  s and the total distance it travels during the 6-s time interval. *Hint:* Plot the path to determine the total distance traveled.

**12–7.** A particle moves along a straight line such that its position is defined by  $s = (t^2 - 6t + 5)$  m. Determine the average velocity, the average speed, and the acceleration of the particle when  $t = 6$  s.

**\*12–8.** A particle is moving along a straight line such that its position is defined by  $s = (10t^2 + 20)$  mm, where  $t$  is in seconds. Determine (a) the displacement of the particle during the time interval from  $t = 1$  s to  $t = 5$  s, (b) the average velocity of the particle during this time interval, and (c) the acceleration when  $t = 1$  s.

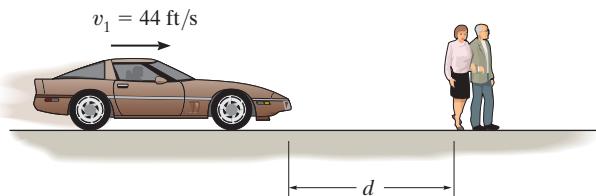
**12–9.** The acceleration of a particle as it moves along a straight line is given by  $a = (2t - 1)$  m/s<sup>2</sup>, where  $t$  is in seconds. If  $s = 1$  m and  $v = 2$  m/s when  $t = 0$ , determine the particle's velocity and position when  $t = 6$  s. Also, determine the total distance the particle travels during this time period.

**12–10.** A particle moves along a straight line with an acceleration of  $a = 5/(3s^{1/3} + s^{5/2})$  m/s<sup>2</sup>, where  $s$  is in meters. Determine the particle's velocity when  $s = 2$  m, if it starts from rest when  $s = 1$  m. Use a numerical method to evaluate the integral.

**12–11.** A particle travels along a straight-line path such that in 4 s it moves from an initial position  $s_A = -8$  m to a position  $s_B = +3$  m. Then in another 5 s it moves from  $s_B$  to  $s_C = -6$  m. Determine the particle's average velocity and average speed during the 9-s time interval.

**\*12–12.** Traveling with an initial speed of 70 km/h, a car accelerates at 6000 km/h<sup>2</sup> along a straight road. How long will it take to reach a speed of 120 km/h? Also, through what distance does the car travel during this time?

**12–13.** Tests reveal that a normal driver takes about 0.75 s before he or she can *react* to a situation to avoid a collision. It takes about 3 s for a driver having 0.1% alcohol in his system to do the same. If such drivers are traveling on a straight road at 30 mph (44 ft/s) and their cars can decelerate at 2 ft/s<sup>2</sup>, determine the shortest stopping distance  $d$  for each from the moment they see the pedestrians. *Moral:* If you must drink, please don't drive!



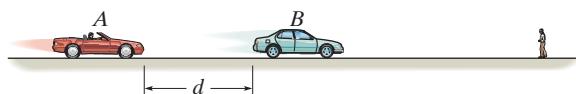
Prob. 12–13

**12-14.** The position of a particle along a straight-line path is defined by  $s = (t^3 - 6t^2 - 15t + 7)$  ft, where  $t$  is in seconds. Determine the total distance traveled when  $t = 10$  s. What are the particle's average velocity, average speed, and the instantaneous velocity and acceleration at this time?

**12-15.** A particle is moving with a velocity of  $v_0$  when  $s = 0$  and  $t = 0$ . If it is subjected to a deceleration of  $a = -kv^3$ , where  $k$  is a constant, determine its velocity and position as functions of time.

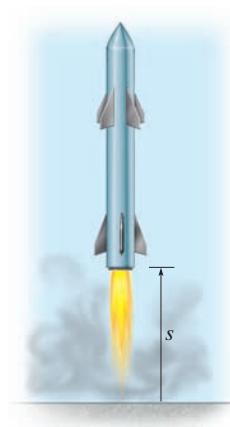
**\*12-16.** A particle is moving along a straight line with an initial velocity of 6 m/s when it is subjected to a deceleration of  $a = (-1.5v^{1/2})$  m/s<sup>2</sup>, where  $v$  is in m/s. Determine how far it travels before it stops. How much time does this take?

**12-17.** Car *B* is traveling a distance  $d$  ahead of car *A*. Both cars are traveling at 60 ft/s when the driver of *B* suddenly applies the brakes, causing his car to decelerate at 12 ft/s<sup>2</sup>. It takes the driver of car *A* 0.75 s to react (this is the normal reaction time for drivers). When he applies his brakes, he decelerates at 15 ft/s<sup>2</sup>. Determine the minimum distance  $d$  between the cars so as to avoid a collision.



Prob. 12-17

**12-18.** The acceleration of a rocket traveling upward is given by  $a = (6 + 0.02s)$  m/s<sup>2</sup>, where  $s$  is in meters. Determine the time needed for the rocket to reach an altitude of  $s = 100$  m. Initially,  $v = 0$  and  $s = 0$  when  $t = 0$ .



Prob. 12-18

**12-19.** A train starts from rest at station *A* and accelerates at 0.5 m/s<sup>2</sup> for 60 s. Afterwards it travels with a constant velocity for 15 min. It then decelerates at 1 m/s<sup>2</sup> until it is brought to rest at station *B*. Determine the distance between the stations.

**\*12-20.** The velocity of a particle traveling along a straight line is  $v = (3t^2 - 6t)$  ft/s, where  $t$  is in seconds. If  $s = 4$  ft when  $t = 0$ , determine the position of the particle when  $t = 4$  s. What is the total distance traveled during the time interval  $t = 0$  to  $t = 4$  s? Also, what is the acceleration when  $t = 2$  s?

**12-21.** A freight train travels at  $v = 60(1 - e^{-t})$  ft/s, where  $t$  is the elapsed time in seconds. Determine the distance traveled in three seconds, and the acceleration at this time.



Prob. 12-21

**12-22.** A sandbag is dropped from a balloon which is ascending vertically at a constant speed of 6 m/s. If the bag is released with the same upward velocity of 6 m/s when  $t = 0$  and hits the ground when  $t = 8$  s, determine the speed of the bag as it hits the ground and the altitude of the balloon at this instant.

**12-23.** A particle is moving along a straight line such that its acceleration is defined as  $a = (-2v)$  m/s<sup>2</sup>, where  $v$  is in meters per second. If  $v = 20$  m/s when  $s = 0$  and  $t = 0$ , determine the particle's position, velocity, and acceleration as functions of time.

**\*12-24.** The acceleration of a particle traveling along a straight line is  $a = \frac{1}{4}s^{1/2}$  m/s<sup>2</sup>, where  $s$  is in meters. If  $v = 0$ ,  $s = 1$  m when  $t = 0$ , determine the particle's velocity at  $s = 2$  m.

**12-25.** If the effects of atmospheric resistance are accounted for, a freely falling body has an acceleration defined by the equation  $a = 9.81[1 - v^2(10^{-4})]$  m/s<sup>2</sup>, where  $v$  is in m/s and the positive direction is downward. If the body is released from rest at a *very high altitude*, determine (a) the velocity when  $t = 5$  s, and (b) the body's terminal or maximum attainable velocity (as  $t \rightarrow \infty$ ).

**12-26.** The acceleration of a particle along a straight line is defined by  $a = (2t - 9)$  m/s<sup>2</sup>, where  $t$  is in seconds. At  $t = 0$ ,  $s = 1$  m and  $v = 10$  m/s. When  $t = 9$  s, determine (a) the particle's position, (b) the total distance traveled, and (c) the velocity.

**12-27.** When a particle falls through the air, its initial acceleration  $a = g$  diminishes until it is zero, and thereafter it falls at a constant or terminal velocity  $v_f$ . If this variation of the acceleration can be expressed as  $a = (g/v_f^2)(v_f^2 - v^2)$ , determine the time needed for the velocity to become  $v = v_f/2$ . Initially the particle falls from rest.

**\*12-28.** Two particles  $A$  and  $B$  start from rest at the origin  $s = 0$  and move along a straight line such that  $a_A = (6t - 3)$  ft/s<sup>2</sup> and  $a_B = (12t^2 - 8)$  ft/s<sup>2</sup>, where  $t$  is in seconds. Determine the distance between them when  $t = 4$  s and the total distance each has traveled in  $t = 4$  s.

**12-29.** A ball  $A$  is thrown vertically upward from the top of a 30-m-high building with an initial velocity of 5 m/s. At the same instant another ball  $B$  is thrown upward from the ground with an initial velocity of 20 m/s. Determine the height from the ground and the time at which they pass.

**12-30.** A sphere is fired downwards into a medium with an initial speed of 27 m/s. If it experiences a deceleration of  $a = (-6t)$  m/s<sup>2</sup>, where  $t$  is in seconds, determine the distance traveled before it stops.

**12-31.** The velocity of a particle traveling along a straight line is  $v = v_0 - ks$ , where  $k$  is constant. If  $s = 0$  when  $t = 0$ , determine the position and acceleration of the particle as a function of time.

**\*12-32.** Ball  $A$  is thrown vertically upwards with a velocity of  $v_0$ . Ball  $B$  is thrown upwards from the same point with the same velocity  $t$  seconds later. Determine the elapsed time  $t < 2v_0/g$  from the instant ball  $A$  is thrown to when the balls pass each other, and find the velocity of each ball at this instant.

**12-33.** As a body is projected to a high altitude above the earth's *surface*, the variation of the acceleration of gravity with respect to altitude  $y$  must be taken into account. Neglecting air resistance, this acceleration is determined from the formula  $a = -g_0[R^2/(R + y)^2]$ , where  $g_0$  is the constant gravitational acceleration at sea level,  $R$  is the radius of the earth, and the positive direction is measured upward. If  $g_0 = 9.81$  m/s<sup>2</sup> and  $R = 6356$  km, determine the minimum initial velocity (escape velocity) at which a projectile should be shot vertically from the earth's surface so that it does not fall back to the earth. *Hint:* This requires that  $v = 0$  as  $y \rightarrow \infty$ .

**12-34.** Accounting for the variation of gravitational acceleration  $a$  with respect to altitude  $y$  (see Prob. 12-33), derive an equation that relates the velocity of a freely falling particle to its altitude. Assume that the particle is released from rest at an altitude  $y_0$  from the earth's surface. With what velocity does the particle strike the earth if it is released from rest at an altitude  $y_0 = 500$  km? Use the numerical data in Prob. 12-33.

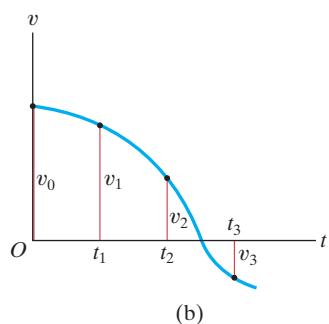
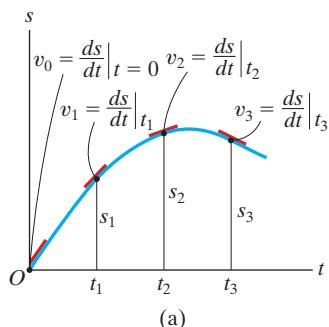
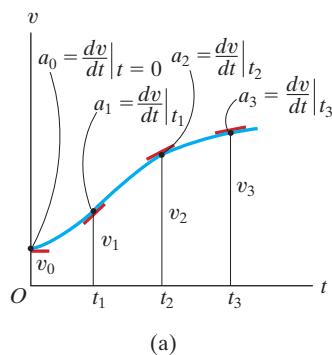


Fig. 12-7



(a)

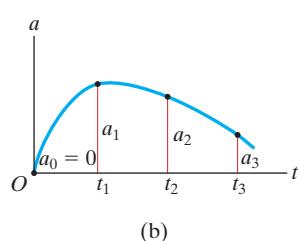


Fig. 12-8

## 12.3 Rectilinear Kinematics: Erratic Motion

When a particle has erratic or changing motion then its position, velocity, and acceleration *cannot* be described by a single continuous mathematical function along the entire path. Instead, a series of functions will be required to specify the motion at different intervals. For this reason, it is convenient to represent the motion as a graph. If a graph of the motion that relates any two of the variables  $s, v, a, t$  can be drawn, then this graph can be used to construct subsequent graphs relating two other variables since the variables are related by the differential relationships  $v = ds/dt$ ,  $a = dv/dt$ , or  $a \, ds = v \, dv$ . Several situations occur frequently.

**The  $s$ - $t$ ,  $v$ - $t$ , and  $a$ - $t$  Graphs.** To construct the  $v$ - $t$  graph given the  $s$ - $t$  graph, Fig. 12-7a, the equation  $v = ds/dt$  should be used, since it relates the variables  $s$  and  $t$  to  $v$ . This equation states that

$$\frac{ds}{dt} = v$$

slope of  $s$ - $t$  graph = velocity

For example, by measuring the slope on the  $s$ - $t$  graph when  $t = t_1$ , the velocity is  $v_1$ , which is plotted in Fig. 12-7b. The  $v$ - $t$  graph can be constructed by plotting this and other values at each instant.

The  $a$ - $t$  graph can be constructed from the  $v$ - $t$  graph in a similar manner, Fig. 12-8, since

$$\frac{dv}{dt} = a$$

slope of  $v$ - $t$  graph = acceleration

Examples of various measurements are shown in Fig. 12-8a and plotted in Fig. 12-8b.

If the  $s$ - $t$  curve for each interval of motion can be expressed by a mathematical function  $s = s(t)$ , then the equation of the  $v$ - $t$  graph for the same interval can be obtained by differentiating this function with respect to time since  $v = ds/dt$ . Likewise, the equation of the  $a$ - $t$  graph for the same interval can be determined by differentiating  $v = v(t)$  since  $a = dv/dt$ . Since differentiation reduces a polynomial of degree  $n$  to that of degree  $n - 1$ , then if the  $s$ - $t$  graph is parabolic (a second-degree curve), the  $v$ - $t$  graph will be a sloping line (a first-degree curve), and the  $a$ - $t$  graph will be a constant or a horizontal line (a zero-degree curve).

If the  $a-t$  graph is given, Fig. 12-9a, the  $v-t$  graph may be constructed using  $a = dv/dt$ , written as

$$\Delta v = \int a dt$$

change in velocity = area under  $a-t$  graph

Hence, to construct the  $v-t$  graph, we begin with the particle's initial velocity  $v_0$  and then add to this small increments of area ( $\Delta v$ ) determined from the  $a-t$  graph. In this manner successive points,  $v_1 = v_0 + \Delta v$ , etc., for the  $v-t$  graph are determined, Fig. 12-9b. Notice that an algebraic addition of the area increments of the  $a-t$  graph is necessary, since areas lying above the  $t$  axis correspond to an increase in  $v$  ("positive" area), whereas those lying below the axis indicate a decrease in  $v$  ("negative" area).

Similarly, if the  $v-t$  graph is given, Fig. 12-10a, it is possible to determine the  $s-t$  graph using  $v = ds/dt$ , written as

$$\Delta s = \int v dt$$

displacement = area under  $v-t$  graph

In the same manner as stated above, we begin with the particle's initial position  $s_0$  and add (algebraically) to this small area increments  $\Delta s$  determined from the  $v-t$  graph, Fig. 12-10b.

If segments of the  $a-t$  graph can be described by a series of equations, then each of these equations can be *integrated* to yield equations describing the corresponding segments of the  $v-t$  graph. In a similar manner, the  $s-t$  graph can be obtained by integrating the equations which describe the segments of the  $v-t$  graph. As a result, if the  $a-t$  graph is linear (a first-degree curve), integration will yield a  $v-t$  graph that is parabolic (a second-degree curve) and an  $s-t$  graph that is cubic (third-degree curve).

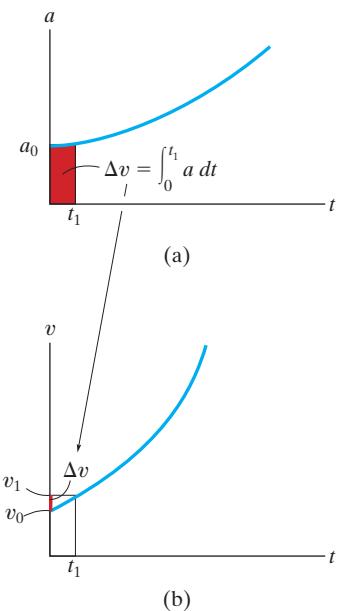


Fig. 12-9

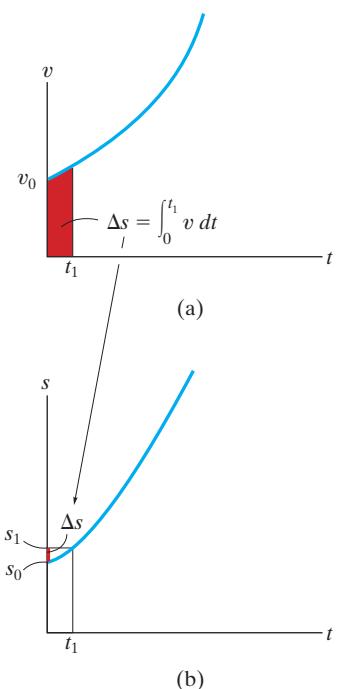
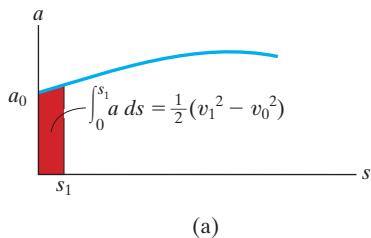
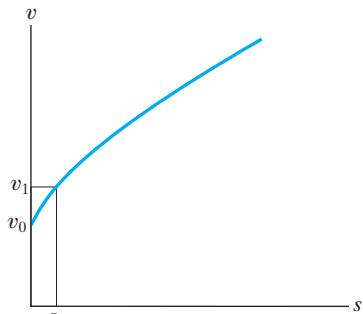


Fig. 12-10



(a)



(b)

**Fig. 12-11**

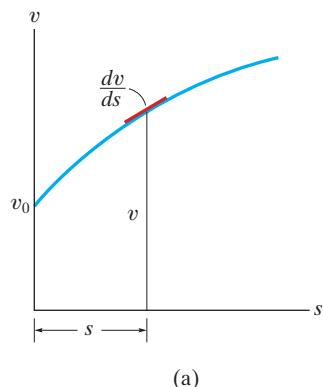
**The  $v$ - $s$  and  $a$ - $s$  Graphs.** If the  $a$ - $s$  graph can be constructed, then points on the  $v$ - $s$  graph can be determined by using  $v \, dv = a \, ds$ . Integrating this equation between the limits  $v = v_0$  at  $s = s_0$  and  $v = v_1$  at  $s = s_1$ , we have,

$$\frac{1}{2}(v_1^2 - v_0^2) = \int_{s_0}^{s_1} a \, ds$$

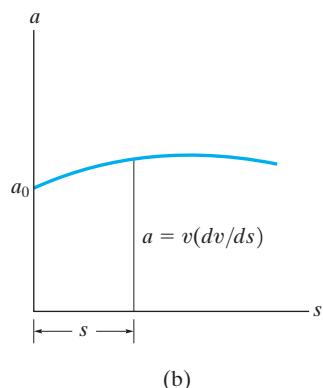
area under  
 $a$ - $s$  graph

Therefore, if the red area in Fig. 12-11a is determined, and the initial velocity  $v_0$  at  $s_0 = 0$  is known, then  $v_1 = (2 \int_0^{s_1} a \, ds + v_0^2)^{1/2}$ , Fig. 12-11b. Successive points on the  $v$ - $s$  graph can be constructed in this manner.

If the  $v$ - $s$  graph is known, the acceleration  $a$  at any position  $s$  can be determined using  $a \, ds = v \, dv$ , written as



(a)



(b)

**Fig. 12-12**

$$a = v \left( \frac{dv}{ds} \right)$$

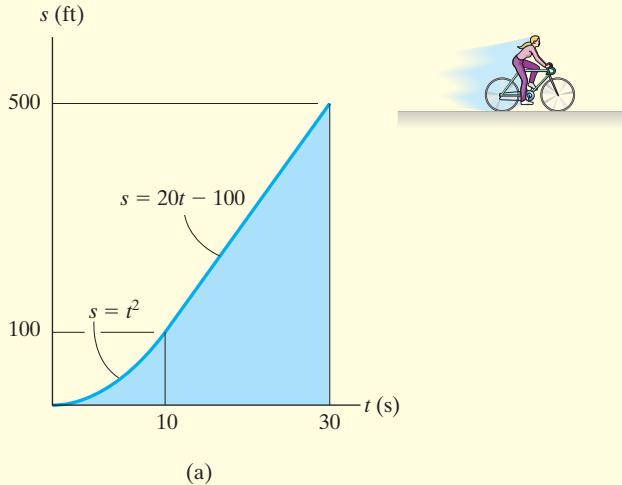
velocity times  
acceleration = slope of  
 $v$ - $s$  graph

Thus, at any point  $(s, v)$  in Fig. 12-12a, the slope  $dv/ds$  of the  $v$ - $s$  graph is measured. Then with  $v$  and  $dv/ds$  known, the value of  $a$  can be calculated, Fig. 12-12b.

The  $v$ - $s$  graph can also be constructed from the  $a$ - $s$  graph, or vice versa, by approximating the known graph in various intervals with mathematical functions,  $v = f(s)$  or  $a = g(s)$ , and then using  $a \, ds = v \, dv$  to obtain the other graph.

**EXAMPLE | 12.6**

A bicycle moves along a straight road such that its position is described by the graph shown in Fig. 12–13a. Construct the  $v$ - $t$  and  $a$ - $t$  graphs for  $0 \leq t \leq 30$  s.

**SOLUTION**

**v-t Graph.** Since  $v = ds/dt$ , the  $v$ - $t$  graph can be determined by differentiating the equations defining the  $s$ - $t$  graph, Fig. 12–13a. We have

$$0 \leq t < 10 \text{ s}; \quad s = (t^2) \text{ ft} \quad v = \frac{ds}{dt} = (2t) \text{ ft/s}$$

$$10 \text{ s} < t \leq 30 \text{ s}; \quad s = (20t - 100) \text{ ft} \quad v = \frac{ds}{dt} = 20 \text{ ft/s}$$

The results are plotted in Fig. 12–13b. We can also obtain specific values of  $v$  by measuring the *slope* of the  $s$ - $t$  graph at a given instant. For example, at  $t = 20$  s, the slope of the  $s$ - $t$  graph is determined from the straight line from 10 s to 30 s, i.e.,

$$t = 20 \text{ s}; \quad v = \frac{\Delta s}{\Delta t} = \frac{500 \text{ ft} - 100 \text{ ft}}{30 \text{ s} - 10 \text{ s}} = 20 \text{ ft/s}$$

**a-t Graph.** Since  $a = dv/dt$ , the  $a$ - $t$  graph can be determined by differentiating the equations defining the lines of the  $v$ - $t$  graph. This yields

$$0 \leq t < 10 \text{ s}; \quad v = (2t) \text{ ft/s} \quad a = \frac{dv}{dt} = 2 \text{ ft/s}^2$$

$$10 < t \leq 30 \text{ s}; \quad v = 20 \text{ ft/s} \quad a = \frac{dv}{dt} = 0$$

The results are plotted in Fig. 12–13c.

**NOTE:** Show that  $a = 2 \text{ ft/s}^2$  when  $t = 5$  s by measuring the slope of the  $v$ - $t$  graph.

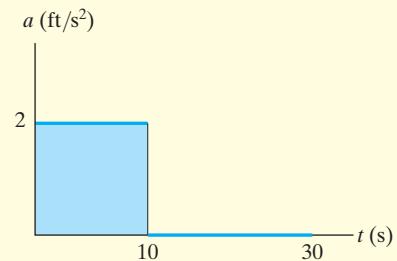
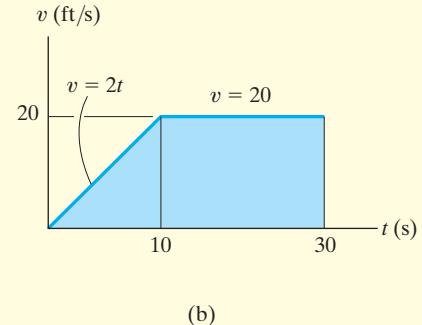
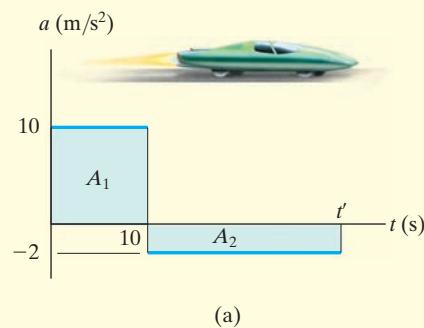


Fig. 12–13

## EXAMPLE | 12.7



The car in Fig. 12-14a starts from rest and travels along a straight track such that it accelerates at  $10 \text{ m/s}^2$  for 10 s, and then decelerates at  $-2 \text{ m/s}^2$ . Draw the  $v$ - $t$  and  $s$ - $t$  graphs and determine the time  $t'$  needed to stop the car. How far has the car traveled?

## SOLUTION

**v-t Graph.** Since  $dv = a dt$ , the  $v$ - $t$  graph is determined by integrating the straight-line segments of the  $a$ - $t$  graph. Using the *initial condition*  $v = 0$  when  $t = 0$ , we have

$$0 \leq t < 10 \text{ s}; \quad a = (10) \text{ m/s}^2; \quad \int_0^v dv = \int_0^t 10 dt, \quad v = 10t$$

When  $t = 10 \text{ s}$ ,  $v = 10(10) = 100 \text{ m/s}$ . Using this as the *initial condition* for the next time period, we have

$$10 \text{ s} < t \leq t'; \quad a = (-2) \text{ m/s}^2; \quad \int_{100 \text{ m/s}}^v dv = \int_{10 \text{ s}}^t -2 dt, \quad v = (-2t + 120) \text{ m/s}$$

When  $t = t'$  we require  $v = 0$ . This yields, Fig. 12-14b,

$$t' = 60 \text{ s} \quad \text{Ans.}$$

A more direct solution for  $t'$  is possible by realizing that the area under the  $a$ - $t$  graph is equal to the change in the car's velocity. We require  $\Delta v = 0 = A_1 + A_2$ , Fig. 12-14a. Thus

$$0 = 10 \text{ m/s}^2(10 \text{ s}) + (-2 \text{ m/s}^2)(t' - 10 \text{ s})$$

$$t' = 60 \text{ s} \quad \text{Ans.}$$

**s-t Graph.** Since  $ds = v dt$ , integrating the equations of the  $v$ - $t$  graph yields the corresponding equations of the  $s$ - $t$  graph. Using the *initial condition*  $s = 0$  when  $t = 0$ , we have

$$0 \leq t \leq 10 \text{ s}; \quad v = (10t) \text{ m/s}; \quad \int_0^s ds = \int_0^t 10t dt, \quad s = (5t^2) \text{ m}$$

When  $t = 10 \text{ s}$ ,  $s = 5(10)^2 = 500 \text{ m}$ . Using this *initial condition*,

$$10 \text{ s} \leq t \leq 60 \text{ s}; \quad v = (-2t + 120) \text{ m/s}; \quad \int_{500 \text{ m}}^s ds = \int_{10 \text{ s}}^t (-2t + 120) dt$$

$$s - 500 = -t^2 + 120t - [-(10)^2 + 120(10)]$$

$$s = (-t^2 + 120t - 600) \text{ m}$$

When  $t' = 60 \text{ s}$ , the position is

$$s = -(60)^2 + 120(60) - 600 = 3000 \text{ m} \quad \text{Ans.}$$

The  $s$ - $t$  graph is shown in Fig. 12-14c.

**NOTE:** A direct solution for  $s$  is possible when  $t' = 60 \text{ s}$ , since the *triangular area* under the  $v$ - $t$  graph would yield the displacement  $\Delta s = s - 0$  from  $t = 0$  to  $t' = 60 \text{ s}$ . Hence,

$$\Delta s = \frac{1}{2}(60 \text{ s})(100 \text{ m/s}) = 3000 \text{ m} \quad \text{Ans.}$$

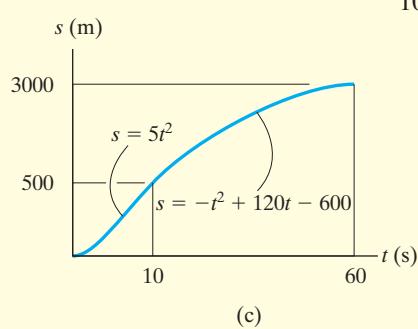
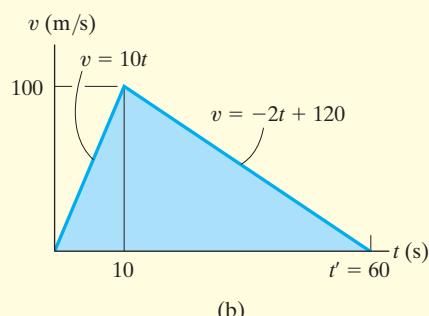


Fig. 12-14

**EXAMPLE | 12.8**

The  $v$ - $s$  graph describing the motion of a motorcycle is shown in Fig. 12–15a. Construct the  $a$ - $s$  graph of the motion and determine the time needed for the motorcycle to reach the position  $s = 400$  ft.

**SOLUTION**

**a-s Graph.** Since the equations for segments of the  $v$ - $s$  graph are given, the  $a$ - $s$  graph can be determined using  $a \, ds = v \, dv$ .

$$0 \leq s < 200 \text{ ft}; \quad v = (0.2s + 10) \text{ ft/s}$$

$$a = v \frac{dv}{ds} = (0.2s + 10) \frac{d}{ds}(0.2s + 10) = 0.04s + 2$$

$$200 \text{ ft} < s \leq 400 \text{ ft}; \quad v = 50 \text{ ft/s}$$

$$a = v \frac{dv}{ds} = (50) \frac{d}{ds}(50) = 0$$

The results are plotted in Fig. 12–15b.

**Time.** The time can be obtained using the  $v$ - $s$  graph and  $v = ds/dt$ , because this equation relates  $v$ ,  $s$ , and  $t$ . For the first segment of motion,  $s = 0$  when  $t = 0$ , so

$$0 \leq s < 200 \text{ ft}; \quad v = (0.2s + 10) \text{ ft/s}; \quad dt = \frac{ds}{v} = \frac{ds}{0.2s + 10}$$

$$\int_0^t dt = \int_0^s \frac{ds}{0.2s + 10}$$

$$t = (5 \ln(0.2s + 10) - 5 \ln 10) \text{ s}$$

At  $s = 200$  ft,  $t = 5 \ln[0.2(200) + 10] - 5 \ln 10 = 8.05$  s. Therefore, using these initial conditions for the second segment of motion,

$$200 \text{ ft} < s \leq 400 \text{ ft}; \quad v = 50 \text{ ft/s}; \quad dt = \frac{ds}{v} = \frac{ds}{50}$$

$$\int_{8.05 \text{ s}}^t dt = \int_{200 \text{ m}}^s \frac{ds}{50};$$

$$t - 8.05 = \frac{s}{50} - 4; \quad t = \left( \frac{s}{50} + 4.05 \right) \text{ s}$$

Therefore, at  $s = 400$  ft,

$$t = \frac{400}{50} + 4.05 = 12.0 \text{ s} \quad \text{Ans.}$$

**NOTE:** The graphical results can be checked in part by calculating slopes. For example, at  $s = 0$ ,  $a = v(dv/ds) = 10(50 - 10)/200 = 2 \text{ m/s}^2$ . Also, the results can be checked in part by inspection. The  $v$ - $s$  graph indicates the initial increase in velocity (acceleration) followed by constant velocity ( $a = 0$ ).

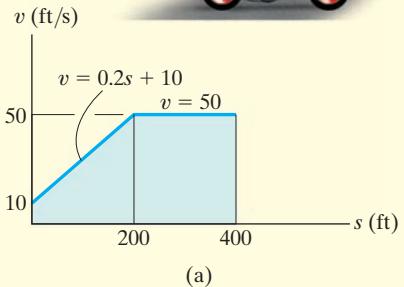
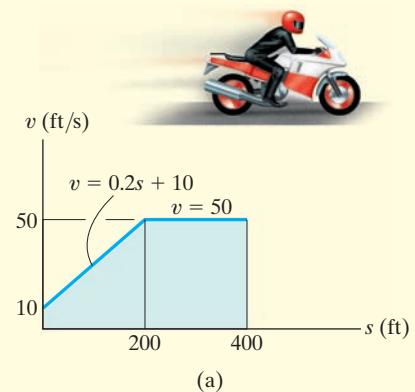
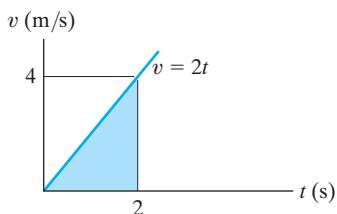


Fig. 12–15

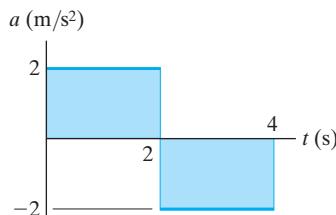
## PRELIMINARY PROBLEM

**P12-2.**

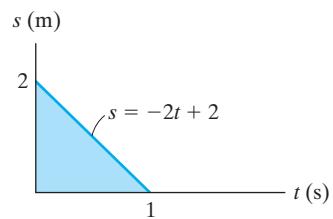
- a) Draw the  $s-t$  and  $a-t$  graphs if  $s = 0$  when  $t = 0$ .



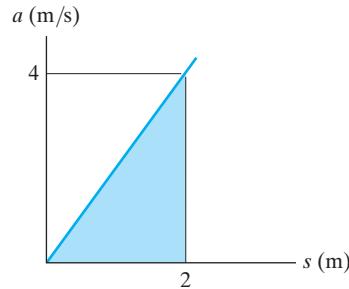
- b) Draw the  $a-t$  and  $v-t$  graphs.



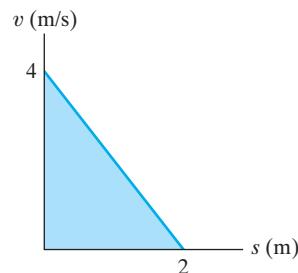
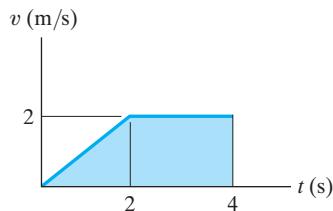
- c) Draw the  $v-t$  and  $s-t$  graphs if  $v = 0$ ,  $s = 0$  when  $t = 0$ .



- f) Determine  $v$  at  $s = 2$  m if  $v = 1$  m/s at  $s = 0$ .



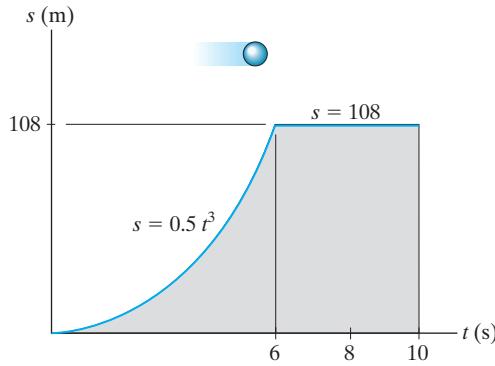
- d) Determine  $s$  and  $a$  when  $t = 3$  s if  $s = 0$  when  $t = 0$ .

**Prob. P12-2**

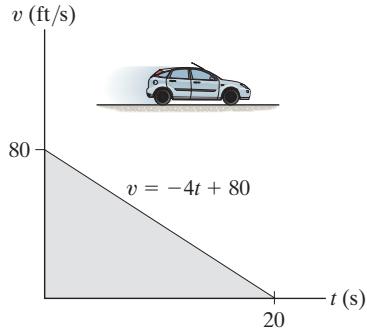
## FUNDAMENTAL PROBLEMS

12

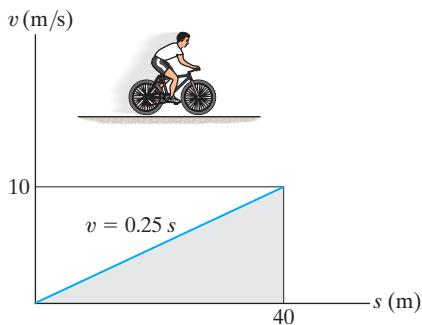
**F12–9.** The particle travels along a straight track such that its position is described by the  $s$ - $t$  graph. Construct the  $v$ - $t$  graph for the same time interval.

**Prob. F12–9**

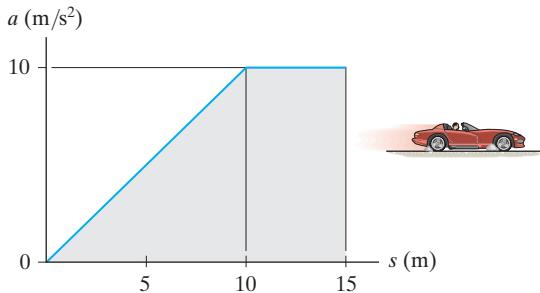
**F12–10.** A van travels along a straight road with a velocity described by the graph. Construct the  $s$ - $t$  and  $a$ - $t$  graphs during the same period. Take  $s = 0$  when  $t = 0$ .

**Prob. F12–10**

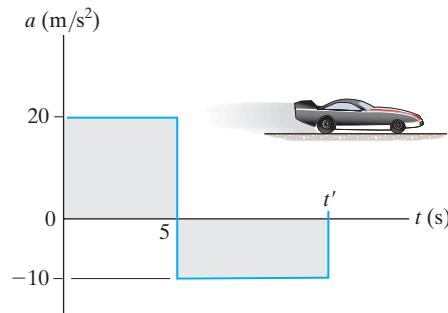
**F12–11.** A bicycle travels along a straight road where its velocity is described by the  $v$ - $s$  graph. Construct the  $a$ - $s$  graph for the same interval.

**Prob. F12–11**

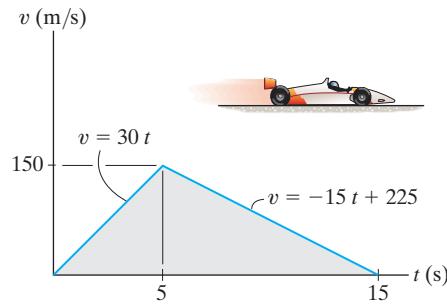
**F12–12.** The sports car travels along a straight road such that its acceleration is described by the graph. Construct the  $v$ - $s$  graph for the same interval and specify the velocity of the car when  $s = 10$  m and  $s = 15$  m.

**Prob. F12–12**

**F12–13.** The dragster starts from rest and has an acceleration described by the graph. Construct the  $v$ - $t$  graph for the time interval  $0 \leq t \leq t'$ , where  $t'$  is the time for the car to come to rest.

**Prob. F12–13**

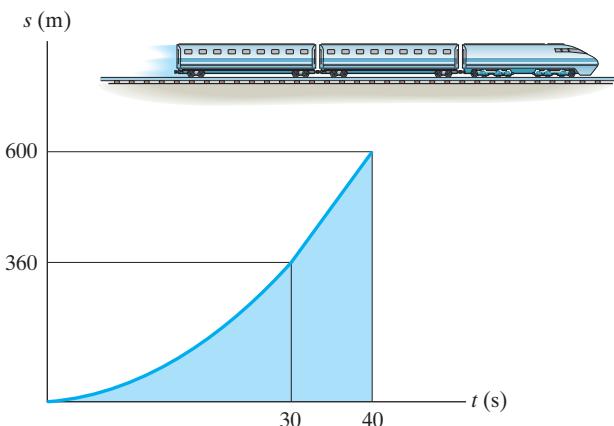
**F12–14.** The dragster starts from rest and has a velocity described by the graph. Construct the  $s$ - $t$  graph during the time interval  $0 \leq t \leq 15$  s. Also, determine the total distance traveled during this time interval.

**Prob. F12–14**

## PROBLEMS

**12-35.** A freight train starts from rest and travels with a constant acceleration of  $0.5 \text{ ft/s}^2$ . After a time  $t'$  it maintains a constant speed so that when  $t = 160 \text{ s}$  it has traveled 2000 ft. Determine the time  $t'$  and draw the  $v-t$  graph for the motion.

\***12-36.** The  $s-t$  graph for a train has been experimentally determined. From the data, construct the  $v-t$  and  $a-t$  graphs for the motion;  $0 \leq t \leq 40 \text{ s}$ . For  $0 \leq t \leq 30 \text{ s}$ , the curve is  $s = (0.4t^2) \text{ m}$ , and then it becomes straight for  $t \geq 30 \text{ s}$ .



Prob. 12-36

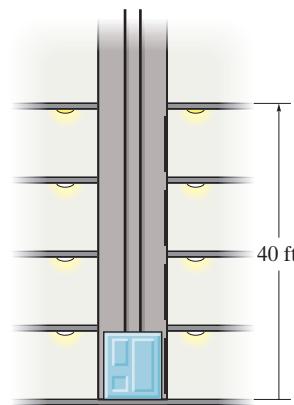
**12-37.** Two rockets start from rest at the same elevation. Rocket A accelerates vertically at  $20 \text{ m/s}^2$  for 12 s and then maintains a constant speed. Rocket B accelerates at  $15 \text{ m/s}^2$  until reaching a constant speed of  $150 \text{ m/s}$ . Construct the  $a-t$ ,  $v-t$ , and  $s-t$  graphs for each rocket until  $t = 20 \text{ s}$ . What is the distance between the rockets when  $t = 20 \text{ s}$ ?

**12-38.** A particle starts from  $s = 0$  and travels along a straight line with a velocity  $v = (t^2 - 4t + 3) \text{ m/s}$ , where  $t$  is in seconds. Construct the  $v-t$  and  $a-t$  graphs for the time interval  $0 \leq t \leq 4 \text{ s}$ .

**12-39.** If the position of a particle is defined by  $s = [2 \sin(\pi/5)t + 4] \text{ m}$ , where  $t$  is in seconds, construct the  $s-t$ ,  $v-t$ , and  $a-t$  graphs for  $0 \leq t \leq 10 \text{ s}$ .

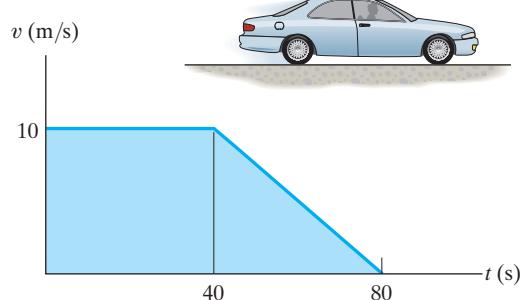
\***12-40.** An airplane starts from rest, travels 5000 ft down a runway, and after uniform acceleration, takes off with a speed of 162 mi/h. It then climbs in a straight line with a uniform acceleration of  $3 \text{ ft/s}^2$  until it reaches a constant speed of 220 mi/h. Draw the  $s-t$ ,  $v-t$ , and  $a-t$  graphs that describe the motion.

**12-41.** The elevator starts from rest at the first floor of the building. It can accelerate at  $5 \text{ ft/s}^2$  and then decelerate at  $2 \text{ ft/s}^2$ . Determine the shortest time it takes to reach a floor 40 ft above the ground. The elevator starts from rest and then stops. Draw the  $a-t$ ,  $v-t$ , and  $s-t$  graphs for the motion.



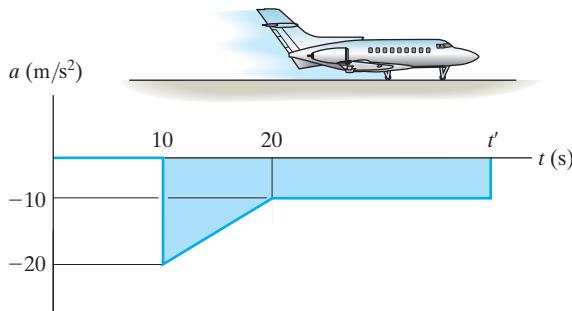
Prob. 12-41

**12-42.** The velocity of a car is plotted as shown. Determine the total distance the car moves until it stops ( $t = 80 \text{ s}$ ). Construct the  $a-t$  graph.



Prob. 12-42

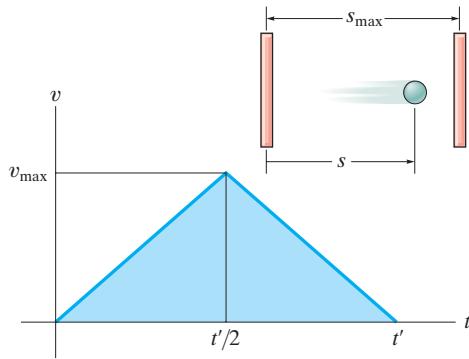
- 12-43.** The motion of a jet plane just after landing on a runway is described by the  $a-t$  graph. Determine the time  $t'$  when the jet plane stops. Construct the  $v-t$  and  $s-t$  graphs for the motion. Here  $s = 0$ , and  $v = 300 \text{ ft/s}$  when  $t = 0$ .



Prob. 12-43

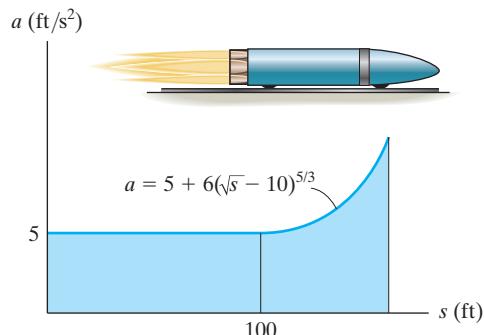
- \*12-44.** The  $v-t$  graph for a particle moving through an electric field from one plate to another has the shape shown in the figure. The acceleration and deceleration that occur are constant and both have a magnitude of  $4 \text{ m/s}^2$ . If the plates are spaced  $200 \text{ mm}$  apart, determine the maximum velocity  $v_{\max}$  and the time  $t'$  for the particle to travel from one plate to the other. Also draw the  $s-t$  graph. When  $t = t'/2$  the particle is at  $s = 100 \text{ mm}$ .

- 12-45.** The  $v-t$  graph for a particle moving through an electric field from one plate to another has the shape shown in the figure, where  $t' = 0.2 \text{ s}$  and  $v_{\max} = 10 \text{ m/s}$ . Draw the  $s-t$  and  $a-t$  graphs for the particle. When  $t = t'/2$  the particle is at  $s = 0.5 \text{ m}$ .



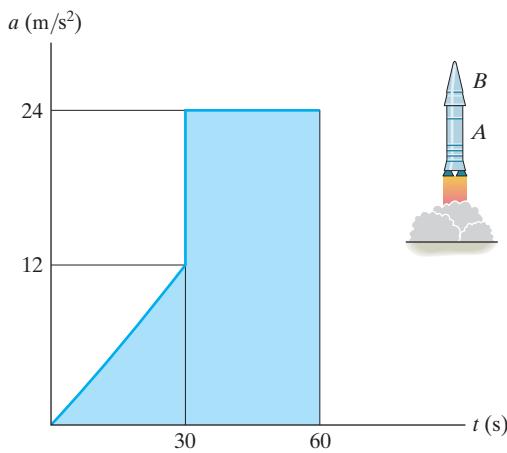
Probs. 12-44/45

- 12-46.** The  $a-s$  graph for a rocket moving along a straight track has been experimentally determined. If the rocket starts at  $s = 0$  when  $v = 0$ , determine its speed when it is at  $s = 75 \text{ ft}$ , and  $125 \text{ ft}$ , respectively. Use Simpson's rule with  $n = 100$  to evaluate  $v$  at  $s = 125 \text{ ft}$ .



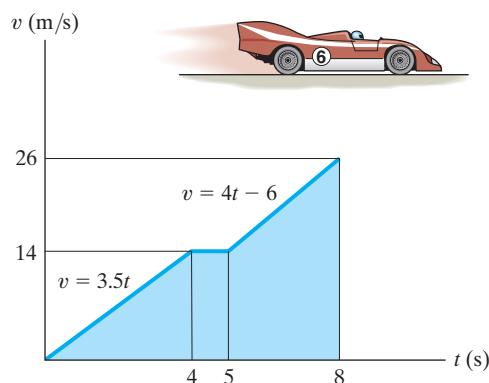
Prob. 12-46

- 12-47.** A two-stage rocket is fired vertically from rest at  $s = 0$  with the acceleration as shown. After  $30 \text{ s}$  the first stage,  $A$ , burns out and the second stage,  $B$ , ignites. Plot the  $v-t$  and  $s-t$  graphs which describe the motion of the second stage for  $0 \leq t \leq 60 \text{ s}$ .



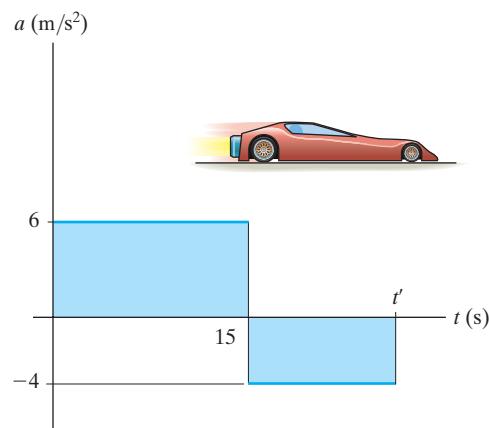
Prob. 12-47

- \*12–48.** The race car starts from rest and travels along a straight road until it reaches a speed of 26 m/s in 8 s as shown on the  $v$ - $t$  graph. The flat part of the graph is caused by shifting gears. Draw the  $a$ - $t$  graph and determine the maximum acceleration of the car.



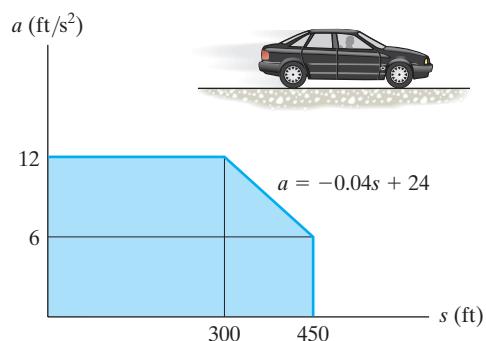
Prob. 12–48

- 12–49.** The jet car is originally traveling at a velocity of 10 m/s when it is subjected to the acceleration shown. Determine the car's maximum velocity and the time  $t'$  when it stops. When  $t = 0, s = 0$ .



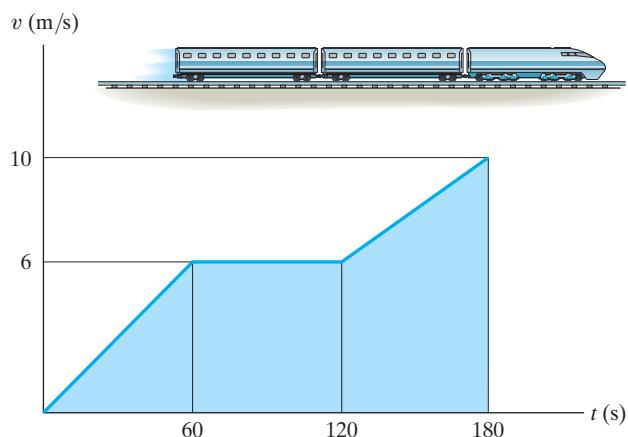
Prob. 12–49

- 12–50.** The car starts from rest at  $s = 0$  and is subjected to an acceleration shown by the  $a$ - $s$  graph. Draw the  $v$ - $s$  graph and determine the time needed to travel 200 ft.



Prob. 12–50

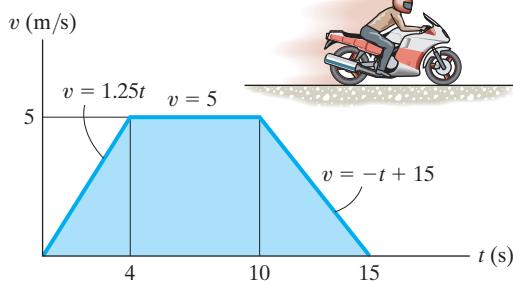
- 12–51.** The  $v$ - $t$  graph for a train has been experimentally determined. From the data, construct the  $s$ - $t$  and  $a$ - $t$  graphs for the motion for  $0 \leq t \leq 180$  s. When  $t = 0, s = 0$ .



Prob. 12–51

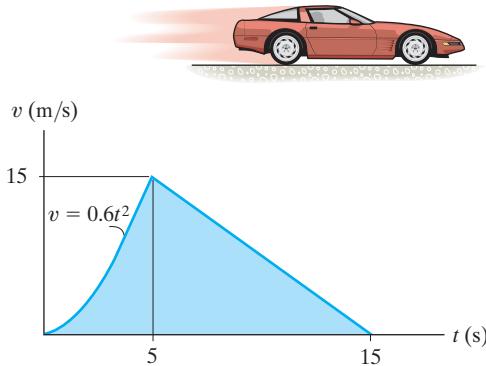
**\*12-52.** A motorcycle starts from rest at  $s = 0$  and travels along a straight road with the speed shown by the  $v-t$  graph. Determine the total distance the motorcycle travels until it stops when  $t = 15$  s. Also plot the  $a-t$  and  $s-t$  graphs.

**12-53.** A motorcycle starts from rest at  $s = 0$  and travels along a straight road with the speed shown by the  $v-t$  graph. Determine the motorcycle's acceleration and position when  $t = 8$  s and  $t = 12$  s.



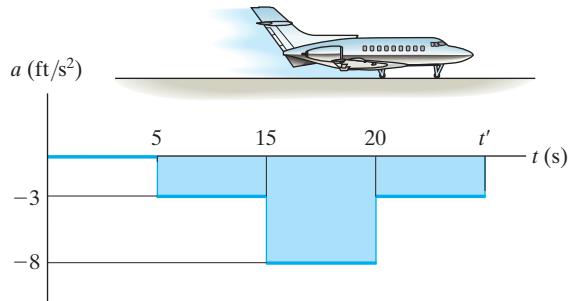
Probs. 12-52/53

**12-54.** The  $v-t$  graph for the motion of a car as it moves along a straight road is shown. Draw the  $s-t$  and  $a-t$  graphs. Also determine the average speed and the distance traveled for the 15-s time interval. When  $t = 0$ ,  $s = 0$ .



Prob. 12-54

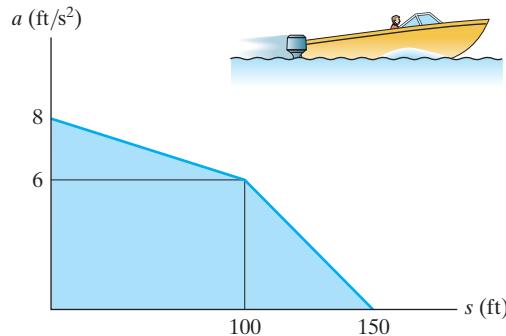
**12-55.** An airplane lands on the straight runway, originally traveling at  $110$  ft/s when  $s = 0$ . If it is subjected to the decelerations shown, determine the time  $t'$  needed to stop the plane and construct the  $s-t$  graph for the motion.



Prob. 12-55

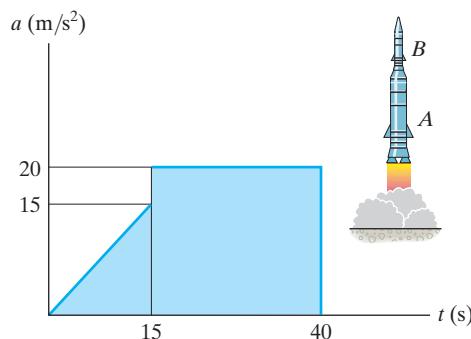
**\*12-56.** Starting from rest at  $s = 0$ , a boat travels in a straight line with the acceleration shown by the  $a-s$  graph. Determine the boat's speed when  $s = 50$  ft,  $100$  ft, and  $150$  ft.

**12-57.** Starting from rest at  $s = 0$ , a boat travels in a straight line with the acceleration shown by the  $a-s$  graph. Construct the  $v-s$  graph.



Probs. 12-56/57

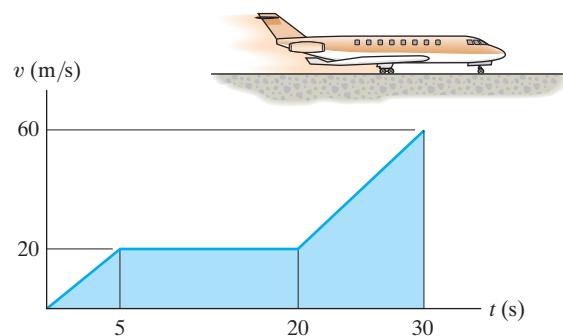
- 12-58.** A two-stage rocket is fired vertically from rest with the acceleration shown. After 15 s the first stage *A* burns out and the second stage *B* ignites. Plot the *v*-*t* and *s*-*t* graphs which describe the motion of the second stage for  $0 \leq t \leq 40$  s.



Prob. 12-58

- 12-62.** If the position of a particle is defined as  $s = (5t - 3t^2)$  ft, where  $t$  is in seconds, construct the *s*-*t*, *v*-*t*, and *a*-*t* graphs for  $0 \leq t \leq 10$  s.

- 12-63.** From experimental data, the motion of a jet plane while traveling along a runway is defined by the *v*-*t* graph. Construct the *s*-*t* and *a*-*t* graphs for the motion. When  $t = 0, s = 0$ .



Prob. 12-63

- 12-59.** The speed of a train during the first minute has been recorded as follows:

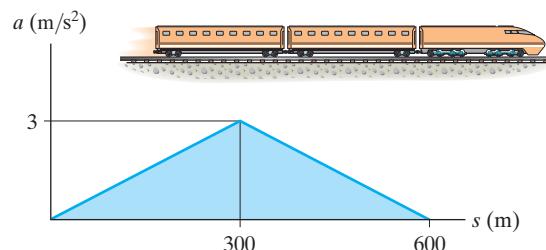
<i>t</i> (s)	0	20	40	60
<i>v</i> (m/s)	0	16	21	24

Plot the *v*-*t* graph, approximating the curve as straight-line segments between the given points. Determine the total distance traveled.

- \*12-60.** A man riding upward in a freight elevator accidentally drops a package off the elevator when it is 100 ft from the ground. If the elevator maintains a constant upward speed of 4 ft/s, determine how high the elevator is from the ground the instant the package hits the ground. Draw the *v*-*t* curve for the package during the time it is in motion. Assume that the package was released with the same upward speed as the elevator.

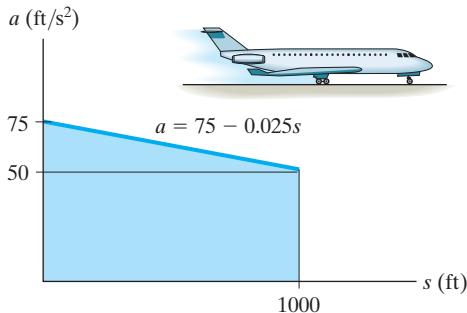
- 12-61.** Two cars start from rest side by side and travel along a straight road. Car *A* accelerates at  $4 \text{ m/s}^2$  for 10 s and then maintains a constant speed. Car *B* accelerates at  $5 \text{ m/s}^2$  until reaching a constant speed of  $25 \text{ m/s}$  and then maintains this speed. Construct the *a*-*t*, *v*-*t*, and *s*-*t* graphs for each car until  $t = 15$  s. What is the distance between the two cars when  $t = 15$  s?

- \*12-64.** The motion of a train is described by the *a*-*s* graph shown. Draw the *v*-*s* graph if  $v = 0$  at  $s = 0$ .

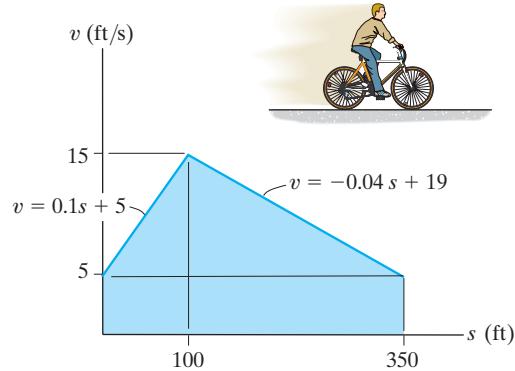


Prob. 12-64

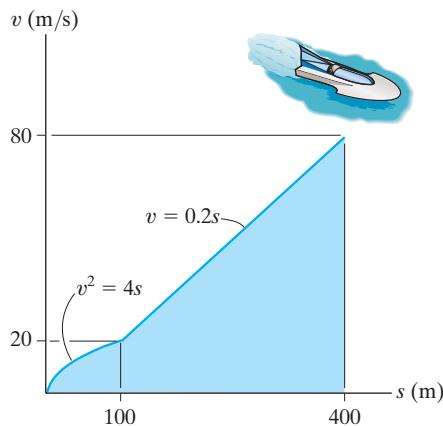
- 12–65.** The jet plane starts from rest at  $s = 0$  and is subjected to the acceleration shown. Determine the speed of the plane when it has traveled 1000 ft. Also, how much time is required for it to travel 1000 ft?

**Prob. 12–65**

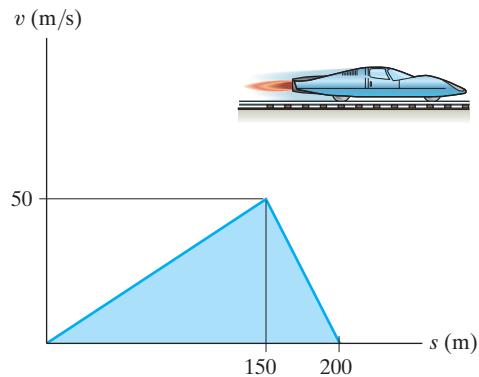
- 12–67.** The  $v$ - $s$  graph of a cyclist traveling along a straight road is shown. Construct the  $a$ - $s$  graph.

**Prob. 12–67**

- 12–66.** The boat travels along a straight line with the speed described by the graph. Construct the  $s$ - $t$  and  $a$ - $s$  graphs. Also, determine the time required for the boat to travel a distance  $s = 400$  m if  $s = 0$  when  $t = 0$ .

**Prob. 12–66**

- \*12–68.** The  $v$ - $s$  graph for a test vehicle is shown. Determine its acceleration when  $s = 100$  m and when  $s = 175$  m.

**Prob. 12–68**

## 12.4 General Curvilinear Motion

*Curvilinear motion* occurs when a particle moves along a curved path. Since this path is often described in three dimensions, vector analysis will be used to formulate the particle's position, velocity, and acceleration.\* In this section the general aspects of curvilinear motion are discussed, and in subsequent sections we will consider three types of coordinate systems often used to analyze this motion.

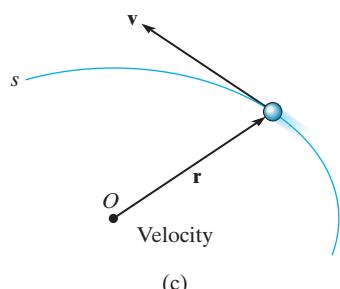
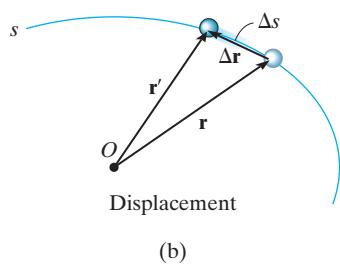
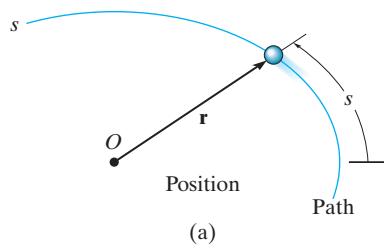


Fig. 12-16

**Position.** Consider a particle located at a point on a space curve defined by the path function  $s(t)$ , Fig. 12-16a. The position of the particle, measured from a fixed point  $O$ , will be designated by the *position vector*  $\mathbf{r} = \mathbf{r}(t)$ . Notice that both the magnitude and direction of this vector will change as the particle moves along the curve.

**Displacement.** Suppose that during a small time interval  $\Delta t$  the particle moves a distance  $\Delta s$  along the curve to a new position, defined by  $\mathbf{r}' = \mathbf{r} + \Delta\mathbf{r}$ , Fig. 12-16b. The *displacement*  $\Delta\mathbf{r}$  represents the change in the particle's position and is determined by vector subtraction; i.e.,  $\Delta\mathbf{r} = \mathbf{r}' - \mathbf{r}$ .

**Velocity.** During the time  $\Delta t$ , the *average velocity* of the particle is

$$\mathbf{v}_{\text{avg}} = \frac{\Delta\mathbf{r}}{\Delta t}$$

The *instantaneous velocity* is determined from this equation by letting  $\Delta t \rightarrow 0$ , and consequently the direction of  $\Delta\mathbf{r}$  approaches the *tangent* to the curve. Hence,  $\mathbf{v} = \lim_{\Delta t \rightarrow 0} (\Delta\mathbf{r}/\Delta t)$  or

$$\boxed{\mathbf{v} = \frac{d\mathbf{r}}{dt}} \quad (12-7)$$

Since  $d\mathbf{r}$  will be tangent to the curve, the *direction* of  $\mathbf{v}$  is also *tangent to the curve*, Fig. 12-16c. The *magnitude* of  $\mathbf{v}$ , which is called the *speed*, is obtained by realizing that the length of the straight line segment  $\Delta\mathbf{r}$  in Fig. 12-16b approaches the arc length  $\Delta s$  as  $\Delta t \rightarrow 0$ , we have  $v = \lim_{\Delta t \rightarrow 0} (\Delta r/\Delta t) = \lim_{\Delta t \rightarrow 0} (\Delta s/\Delta t)$ , or

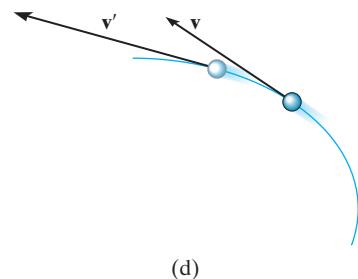
$$\boxed{v = \frac{ds}{dt}} \quad (12-8)$$

Thus, the *speed* can be obtained by differentiating the path function  $s$  with respect to time.

\*A summary of some of the important concepts of vector analysis is given in Appendix B.

**Acceleration.** If the particle has a velocity  $\mathbf{v}$  at time  $t$  and a velocity  $\mathbf{v}' = \mathbf{v} + \Delta\mathbf{v}$  at  $t + \Delta t$ , Fig. 12–16d, then the *average acceleration* of the particle during the time interval  $\Delta t$  is

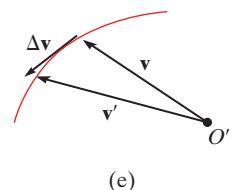
$$\mathbf{a}_{\text{avg}} = \frac{\Delta\mathbf{v}}{\Delta t}$$



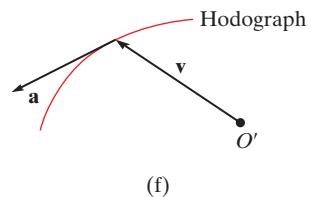
where  $\Delta\mathbf{v} = \mathbf{v}' - \mathbf{v}$ . To study this time rate of change, the two velocity vectors in Fig. 12–16d are plotted in Fig. 12–16e such that their tails are located at the fixed point  $O'$  and their arrowheads touch points on a curve. This curve is called a *hodograph*, and when constructed, it describes the locus of points for the arrowhead of the velocity vector in the same manner as the *path s* describes the locus of points for the arrowhead of the position vector, Fig. 12–16a.

To obtain the *instantaneous acceleration*, let  $\Delta t \rightarrow 0$  in the above equation. In the limit  $\Delta\mathbf{v}$  will approach the *tangent to the hodograph*, and so  $\mathbf{a} = \lim_{\Delta t \rightarrow 0} (\Delta\mathbf{v}/\Delta t)$ , or

$$\mathbf{a} = \frac{d\mathbf{v}}{dt} \quad (12-9)$$



(e)



(f)

Substituting Eq. 12–7 into this result, we can also write

$$\mathbf{a} = \frac{d^2\mathbf{r}}{dt^2}$$

By definition of the derivative,  $\mathbf{a}$  acts *tangent to the hodograph*, Fig. 12–16f, and, in general it is not tangent to the path of motion, Fig. 12–16g. To clarify this point, realize that  $\Delta\mathbf{v}$  and consequently  $\mathbf{a}$  must account for the change made in both the magnitude and direction of the velocity  $\mathbf{v}$  as the particle moves from one point to the next along the path, Fig. 12–16d. However, in order for the particle to follow any curved path, the directional change always “swings” the velocity vector toward the “inside” or “concave side” of the path, and therefore  $\mathbf{a}$  cannot remain tangent to the path. In summary,  $\mathbf{v}$  is always tangent to the path and  $\mathbf{a}$  is always tangent to the hodograph.

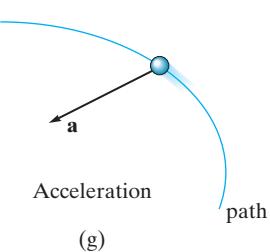
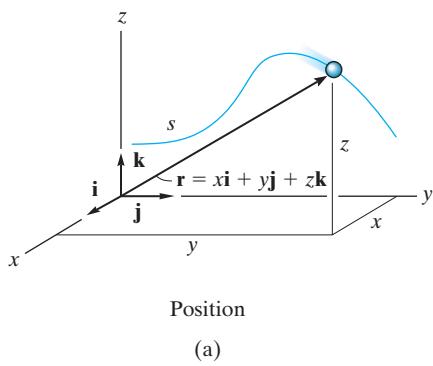


Fig. 12-16

## 12.5 Curvilinear Motion: Rectangular Components

Occasionally the motion of a particle can best be described along a path that can be expressed in terms of its  $x, y, z$  coordinates.



**Position.** If the particle is at point  $(x, y, z)$  on the curved path  $s$  shown in Fig. 12-17a, then its location is defined by the *position vector*

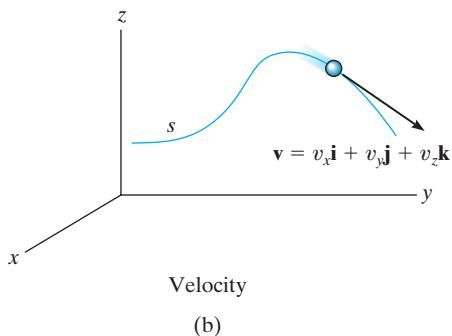
$$\mathbf{r} = x\mathbf{i} + y\mathbf{j} + z\mathbf{k} \quad (12-10)$$

When the particle moves, the  $x, y, z$  components of  $\mathbf{r}$  will be functions of time; i.e.,  $x = x(t), y = y(t), z = z(t)$ , so that  $\mathbf{r} = \mathbf{r}(t)$ .

At any instant the *magnitude* of  $\mathbf{r}$  is defined from Eq. B-3 in Appendix B as

$$r = \sqrt{x^2 + y^2 + z^2}$$

And the *direction* of  $\mathbf{r}$  is specified by the unit vector  $\mathbf{u}_r = \mathbf{r}/r$ .



**Velocity.** The first time derivative of  $\mathbf{r}$  yields the velocity of the particle. Hence,

$$\mathbf{v} = \frac{d\mathbf{r}}{dt} = \frac{d}{dt}(x\mathbf{i}) + \frac{d}{dt}(y\mathbf{j}) + \frac{d}{dt}(z\mathbf{k})$$

When taking this derivative, it is necessary to account for changes in *both* the magnitude and direction of each of the vector's components. For example, the derivative of the  $\mathbf{i}$  component of  $\mathbf{r}$  is

$$\frac{d}{dt}(x\mathbf{i}) = \frac{dx}{dt}\mathbf{i} + x\frac{d\mathbf{i}}{dt}$$

The second term on the right side is zero, provided the  $x, y, z$  reference frame is *fixed*, and therefore the *direction* (and the *magnitude*) of  $\mathbf{i}$  does not change with time. Differentiation of the  $\mathbf{j}$  and  $\mathbf{k}$  components may be carried out in a similar manner, which yields the final result,

$$\mathbf{v} = \frac{d\mathbf{r}}{dt} = v_x\mathbf{i} + v_y\mathbf{j} + v_z\mathbf{k} \quad (12-11)$$

where

$$v_x = \dot{x} \quad v_y = \dot{y} \quad v_z = \dot{z} \quad (12-12)$$

Fig. 12-17

The “dot” notation  $\dot{x}, \dot{y}, \dot{z}$  represents the first time derivatives of  $x = x(t)$ ,  $y = y(t)$ ,  $z = z(t)$ , respectively.

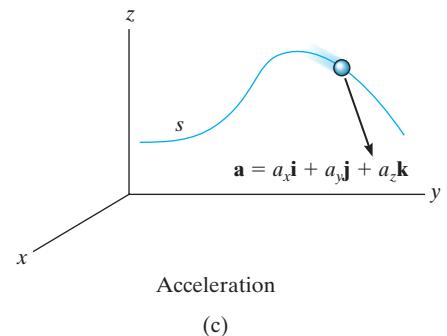
The velocity has a *magnitude* that is found from

$$v = \sqrt{v_x^2 + v_y^2 + v_z^2}$$

and a *direction* that is specified by the unit vector  $\mathbf{u}_v = \mathbf{v}/v$ . As discussed in Sec. 12.4, this direction is *always tangent to the path*, as shown in Fig. 12-17b.

**Acceleration.** The acceleration of the particle is obtained by taking the first time derivative of Eq. 12-11 (or the second time derivative of Eq. 12-10). We have

$$\mathbf{a} = \frac{d\mathbf{v}}{dt} = a_x \mathbf{i} + a_y \mathbf{j} + a_z \mathbf{k} \quad (12-13)$$



where

$$\begin{aligned} a_x &= \dot{v}_x = \ddot{x} \\ a_y &= \dot{v}_y = \ddot{y} \\ a_z &= \dot{v}_z = \ddot{z} \end{aligned} \quad (12-14)$$

Here  $a_x, a_y, a_z$  represent, respectively, the first time derivatives of  $v_x = v_x(t), v_y = v_y(t), v_z = v_z(t)$ , or the second time derivatives of the functions  $x = x(t), y = y(t), z = z(t)$ .

The acceleration has a *magnitude*

$$a = \sqrt{a_x^2 + a_y^2 + a_z^2}$$

and a *direction* specified by the unit vector  $\mathbf{u}_a = \mathbf{a}/a$ . Since  $\mathbf{a}$  represents the time rate of *change* in both the magnitude and direction of the velocity, in general  $\mathbf{a}$  will *not* be tangent to the path, Fig. 12-17c.

## Important Points

- Curvilinear motion can cause changes in *both* the magnitude and direction of the position, velocity, and acceleration vectors.
- The velocity vector is always directed *tangent* to the path.
- In general, the acceleration vector is *not* tangent to the path, but rather, it is tangent to the hodograph.
- If the motion is described using rectangular coordinates, then the components along each of the axes do not change direction, only their magnitude and sense (algebraic sign) will change.
- By considering the component motions, the change in magnitude and direction of the particle's position and velocity are automatically taken into account.

## Procedure for Analysis

### Coordinate System.

- A rectangular coordinate system can be used to solve problems for which the motion can conveniently be expressed in terms of its  $x, y, z$  components.

### Kinematic Quantities.

- Since *rectilinear motion* occurs along *each coordinate axis*, the motion along each axis is found using  $v = ds/dt$  and  $a = dv/dt$ ; or in cases where the motion is not expressed as a function of time, the equation  $a \, ds = v \, dv$  can be used.
- In two dimensions, the equation of the path  $y = f(x)$  can be used to relate the  $x$  and  $y$  components of velocity and acceleration by applying the chain rule of calculus. A review of this concept is given in Appendix C.
- Once the  $x, y, z$  components of  $\mathbf{v}$  and  $\mathbf{a}$  have been determined, the magnitudes of these vectors are found from the Pythagorean theorem, Eq. B-3, and their coordinate direction angles from the components of their unit vectors, Eqs. B-4 and B-5.

**EXAMPLE | 12.9**

At any instant the horizontal position of the weather balloon in Fig. 12–18a is defined by  $x = (8t)$  ft, where  $t$  is in seconds. If the equation of the path is  $y = x^2/10$ , determine the magnitude and direction of the velocity and the acceleration when  $t = 2$  s.

**SOLUTION**

**Velocity.** The velocity component in the  $x$  direction is

$$v_x = \dot{x} = \frac{d}{dt}(8t) = 8 \text{ ft/s} \rightarrow$$

To find the relationship between the velocity components we will use the chain rule of calculus. When  $t = 2$  s,  $x = 8(2) = 16$  ft, Fig. 12–18a, and so

$$v_y = \dot{y} = \frac{d}{dt}(x^2/10) = 2x\dot{x}/10 = 2(16)(8)/10 = 25.6 \text{ ft/s} \uparrow$$

When  $t = 2$  s, the magnitude of velocity is therefore

$$v = \sqrt{(8 \text{ ft/s})^2 + (25.6 \text{ ft/s})^2} = 26.8 \text{ ft/s} \quad \text{Ans.}$$

The direction is tangent to the path, Fig. 12–18b, where

$$\theta_v = \tan^{-1} \frac{v_y}{v_x} = \tan^{-1} \frac{25.6}{8} = 72.6^\circ \quad \text{Ans.}$$

**Acceleration.** The relationship between the acceleration components is determined using the chain rule. (See Appendix C.) We have

$$\begin{aligned} a_x &= \ddot{v}_x = \frac{d}{dt}(\dot{x}) = 0 \\ a_y &= \ddot{v}_y = \frac{d}{dt}(2x\dot{x}/10) = 2(\ddot{x})\dot{x}/10 + 2x(\ddot{x})/10 \\ &= 2(8)^2/10 + 2(16)(0)/10 = 12.8 \text{ ft/s}^2 \uparrow \end{aligned}$$

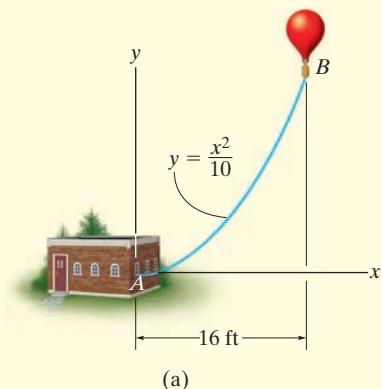
Thus,

$$a = \sqrt{(0)^2 + (12.8)^2} = 12.8 \text{ ft/s}^2 \quad \text{Ans.}$$

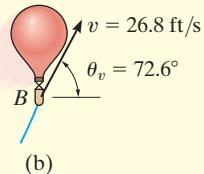
The direction of  $\mathbf{a}$ , as shown in Fig. 12–18c, is

$$\theta_a = \tan^{-1} \frac{12.8}{0} = 90^\circ \quad \text{Ans.}$$

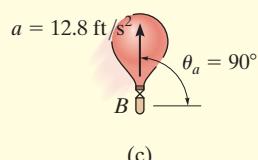
**NOTE:** It is also possible to obtain  $v_y$  and  $a_y$  by first expressing  $y = f(t) = (8t)^2/10 = 6.4t^2$  and then taking successive time derivatives.



(a)



(b)



(c)

**Fig. 12–18**

## EXAMPLE 12.10



(© R.C. Hibbeler)

For a short time, the path of the plane in Fig. 12–19a is described by  $y = (0.001x^2)$  m. If the plane is rising with a constant upward velocity of 10 m/s, determine the magnitudes of the velocity and acceleration of the plane when it reaches an altitude of  $y = 100$  m.

**SOLUTION**

When  $y = 100$  m, then  $100 = 0.001x^2$  or  $x = 316.2$  m. Also, due to constant velocity  $v_y = 10$  m/s, so

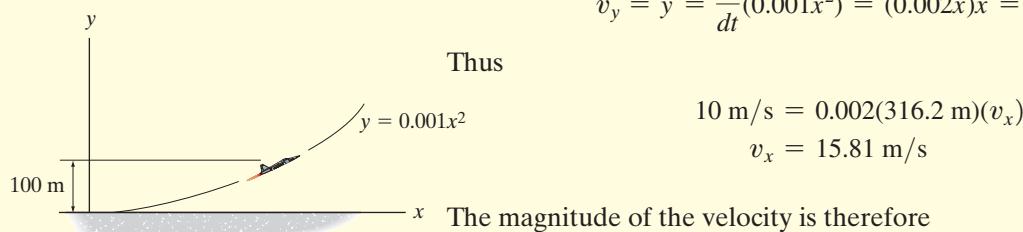
$$y = v_y t; \quad 100 \text{ m} = (10 \text{ m/s}) t \quad t = 10 \text{ s}$$

**Velocity.** Using the chain rule (see Appendix C) to find the relationship between the velocity components, we have

$$y = 0.001x^2$$

$$v_y = \dot{y} = \frac{d}{dt}(0.001x^2) = (0.002x)\dot{x} = 0.002xv_x \quad (1)$$

Thus



(a)

$$10 \text{ m/s} = 0.002(316.2 \text{ m})(v_x)$$

$$v_x = 15.81 \text{ m/s}$$

The magnitude of the velocity is therefore

$$v = \sqrt{v_x^2 + v_y^2} = \sqrt{(15.81 \text{ m/s})^2 + (10 \text{ m/s})^2} = 18.7 \text{ m/s} \quad \text{Ans.}$$

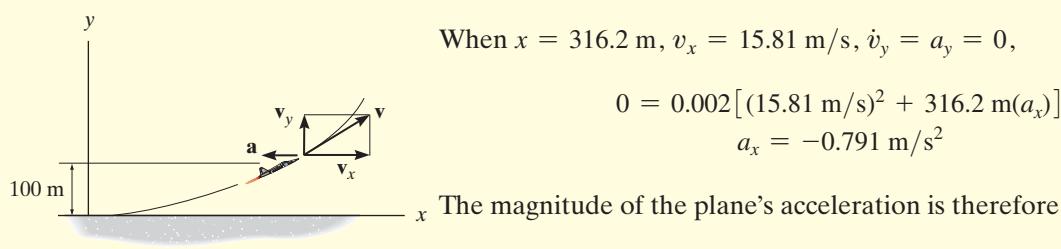
**Acceleration.** Using the chain rule, the time derivative of Eq. (1) gives the relation between the acceleration components.

$$a_y = \dot{v}_y = (0.002\dot{x})\dot{x} + 0.002x(\ddot{x}) = 0.002(v_x^2 + x a_x)$$

When  $x = 316.2$  m,  $v_x = 15.81$  m/s,  $\dot{v}_y = a_y = 0$ ,

$$0 = 0.002[(15.81 \text{ m/s})^2 + 316.2 \text{ m}(a_x)]$$

$$a_x = -0.791 \text{ m/s}^2$$



The magnitude of the plane's acceleration is therefore

$$a = \sqrt{a_x^2 + a_y^2} = \sqrt{(-0.791 \text{ m/s}^2)^2 + (0 \text{ m/s}^2)^2}$$

$$= 0.791 \text{ m/s}^2 \quad \text{Ans.}$$

Fig. 12–19

These results are shown in Fig. 12–19b.

## 12.6 Motion of a Projectile

The free-flight motion of a projectile is often studied in terms of its rectangular components. To illustrate the kinematic analysis, consider a projectile launched at point  $(x_0, y_0)$ , with an initial velocity of  $\mathbf{v}_0$ , having components  $(\mathbf{v}_0)_x$  and  $(\mathbf{v}_0)_y$ , Fig. 12-20. When air resistance is neglected, the only force acting on the projectile is its weight, which causes the projectile to have a *constant downward acceleration* of approximately  $a_c = g = 9.81 \text{ m/s}^2$  or  $g = 32.2 \text{ ft/s}^2$ .\*

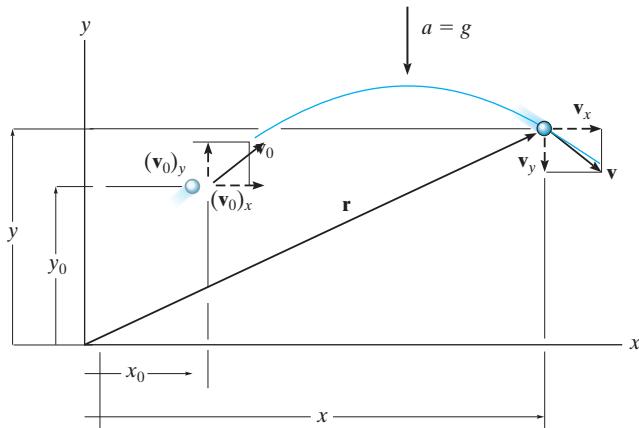
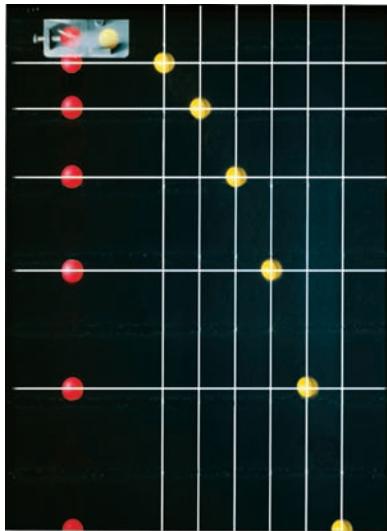


Fig. 12-20



**Horizontal Motion.** Since  $a_x = 0$ , application of the constant acceleration equations, 12-4 to 12-6, yields

$$\begin{aligned} (\pm) \quad v &= v_0 + a_c t & v_x &= (v_0)_x \\ (\pm) \quad x &= x_0 + v_0 t + \frac{1}{2} a_c t^2; & x &= x_0 + (v_0)_x t \\ (\pm) \quad v^2 &= v_0^2 + 2a_c(x - x_0); & v_x &= (v_0)_x \end{aligned}$$

The first and last equations indicate that *the horizontal component of velocity always remains constant during the motion*.

**Vertical Motion.** Since the positive  $y$  axis is directed upward, then  $a_y = -g$ . Applying Eqs. 12-4 to 12-6, we get

$$\begin{aligned} (+\uparrow) \quad v &= v_0 + a_c t; & v_y &= (v_0)_y - gt \\ (+\uparrow) \quad y &= y_0 + v_0 t + \frac{1}{2} a_c t^2; & y &= y_0 + (v_0)_y t - \frac{1}{2} gt^2 \\ (+\uparrow) \quad v^2 &= v_0^2 + 2a_c(y - y_0); & v_y^2 &= (v_0)_y^2 - 2g(y - y_0) \end{aligned}$$

Recall that the last equation can be formulated on the basis of eliminating the time  $t$  from the first two equations, and therefore *only two of the above three equations are independent of one another*.

Each picture in this sequence is taken after the same time interval. The red ball falls from rest, whereas the yellow ball is given a horizontal velocity when released. Both balls accelerate downward at the same rate, and so they remain at the same elevation at any instant. This acceleration causes the difference in elevation between the balls to increase between successive photos. Also, note the horizontal distance between successive photos of the yellow ball is constant since the velocity in the horizontal direction remains constant.  
© R.C. Hibbeler)

\*This assumes that the earth's gravitational field does not vary with altitude.



Once thrown, the basketball follows a parabolic trajectory. (© R.C. Hibbeler)

To summarize, problems involving the motion of a projectile can have at most three unknowns since only three independent equations can be written; that is, *one equation in the horizontal direction and two in the vertical direction*. Once  $v_x$  and  $v_y$  are obtained, the resultant velocity  $\mathbf{v}$ , which is *always tangent to the path*, can be determined by the *vector sum* as shown in Fig. 12–20.

## Procedure for Analysis

### Coordinate System.

- Establish the fixed  $x$ ,  $y$  coordinate axes and sketch the trajectory of the particle. Between any *two points* on the path specify the given problem data and identify the *three unknowns*. In all cases the acceleration of gravity acts downward and equals  $9.81 \text{ m/s}^2$  or  $32.2 \text{ ft/s}^2$ . The particle's initial and final velocities should be represented in terms of their  $x$  and  $y$  components.
- Remember that positive and negative position, velocity, and acceleration components always act in accordance with their associated coordinate directions.

### Kinematic Equations.

- Depending upon the known data and what is to be determined, a choice should be made as to which three of the following four equations should be applied between the two points on the path to obtain the most direct solution to the problem.

### Horizontal Motion.

- The *velocity* in the horizontal or  $x$  direction is *constant*, i.e.,  $v_x = (v_0)_x$ , and

$$x = x_0 + (v_0)_x t$$

### Vertical Motion.

- In the vertical or  $y$  direction *only two* of the following three equations can be used for solution.

$$v_y = (v_0)_y + a_c t$$

$$y = y_0 + (v_0)_y t + \frac{1}{2} a_c t^2$$

$$v_y^2 = (v_0)_y^2 + 2a_c(y - y_0)$$

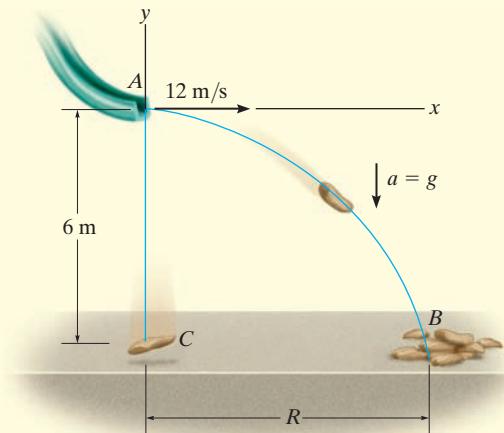


Gravel falling off the end of this conveyor belt follows a path that can be predicted using the equations of constant acceleration. In this way the location of the accumulated pile can be determined. Rectangular coordinates are used for the analysis since the acceleration is only in the vertical direction. (© R.C. Hibbeler)

For example, if the particle's final velocity  $v_y$  is not needed, then the first and third of these equations will not be useful.

**EXAMPLE | 12.11**

A sack slides off the ramp, shown in Fig. 12–21, with a horizontal velocity of 12 m/s. If the height of the ramp is 6 m from the floor, determine the time needed for the sack to strike the floor and the range  $R$  where sacks begin to pile up.

**Fig. 12–21****SOLUTION**

**Coordinate System.** The origin of coordinates is established at the beginning of the path, point  $A$ , Fig. 12–21. The initial velocity of a sack has components  $(v_A)_x = 12 \text{ m/s}$  and  $(v_A)_y = 0$ . Also, between points  $A$  and  $B$  the acceleration is  $a_y = -9.81 \text{ m/s}^2$ . Since  $(v_B)_x = (v_A)_x = 12 \text{ m/s}$ , the three unknowns are  $(v_B)_y$ ,  $R$ , and the time of flight  $t_{AB}$ . Here we do not need to determine  $(v_B)_y$ .

**Vertical Motion.** The vertical distance from  $A$  to  $B$  is known, and therefore we can obtain a direct solution for  $t_{AB}$  by using the equation

$$(+\uparrow) \quad y_B = y_A + (v_A)_y t_{AB} + \frac{1}{2} a_c t_{AB}^2$$

$$-6 \text{ m} = 0 + 0 + \frac{1}{2}(-9.81 \text{ m/s}^2) t_{AB}^2$$

$$t_{AB} = 1.11 \text{ s} \quad \text{Ans.}$$

**Horizontal Motion.** Since  $t_{AB}$  has been calculated,  $R$  is determined as follows:

$$(\pm\rightarrow) \quad x_B = x_A + (v_A)_x t_{AB}$$

$$R = 0 + 12 \text{ m/s} (1.11 \text{ s})$$

$$R = 13.3 \text{ m} \quad \text{Ans.}$$

**NOTE:** The calculation for  $t_{AB}$  also indicates that if a sack were released from rest at  $A$ , it would take the same amount of time to strike the floor at  $C$ , Fig. 12–21.

## EXAMPLE 12.12

The chipping machine is designed to eject wood chips at  $v_O = 25 \text{ ft/s}$  as shown in Fig. 12–22. If the tube is oriented at  $30^\circ$  from the horizontal, determine how high,  $h$ , the chips strike the pile if at this instant they land on the pile 20 ft from the tube.

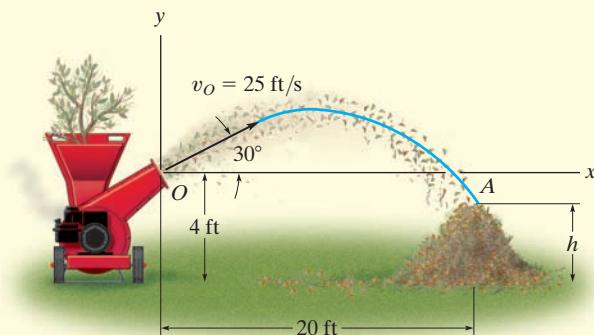


Fig. 12-22

## SOLUTION

**Coordinate System.** When the motion is analyzed between points  $O$  and  $A$ , the three unknowns are the height  $h$ , time of flight  $t_{OA}$ , and vertical component of velocity  $(v_A)_y$ . [Note that  $(v_A)_x = (v_O)_x$ .] With the origin of coordinates at  $O$ , Fig. 12–22, the initial velocity of a chip has components of

$$(v_O)_x = (25 \cos 30^\circ) \text{ ft/s} = 21.65 \text{ ft/s} \rightarrow$$

$$(v_O)_y = (25 \sin 30^\circ) \text{ ft/s} = 12.5 \text{ ft/s} \uparrow$$

Also,  $(v_A)_x = (v_O)_x = 21.65 \text{ ft/s}$  and  $a_y = -32.2 \text{ ft/s}^2$ . Since we do not need to determine  $(v_A)_y$ , we have

**Horizontal Motion.**

$$\begin{aligned} (\pm) \quad x_A &= x_O + (v_O)_x t_{OA} \\ 20 \text{ ft} &= 0 + (21.65 \text{ ft/s}) t_{OA} \\ t_{OA} &= 0.9238 \text{ s} \end{aligned}$$

**Vertical Motion.** Relating  $t_{OA}$  to the initial and final elevations of a chip, we have

$$\begin{aligned} (+\uparrow) \quad y_A &= y_O + (v_O)_y t_{OA} + \frac{1}{2} a_c t_{OA}^2 \\ (h - 4 \text{ ft}) &= 0 + (12.5 \text{ ft/s})(0.9238 \text{ s}) + \frac{1}{2}(-32.2 \text{ ft/s}^2)(0.9238 \text{ s})^2 \\ h &= 1.81 \text{ ft} \end{aligned}$$

Ans.

**NOTE:** We can determine  $(v_A)_y$  by using  $(v_A)_y = (v_O)_y + a_c t_{OA}$ .

**EXAMPLE | 12.13**

The track for this racing event was designed so that riders jump off the slope at  $30^\circ$ , from a height of 1 m. During a race it was observed that the rider shown in Fig. 12–23a remained in mid air for 1.5 s. Determine the speed at which he was traveling off the ramp, the horizontal distance he travels before striking the ground, and the maximum height he attains. Neglect the size of the bike and rider.



(© R.C. Hibbeler)

(a)

**SOLUTION**

**Coordinate System.** As shown in Fig. 12–23b, the origin of the coordinates is established at *A*. Between the end points of the path *AB* the three unknowns are the initial speed  $v_A$ , range  $R$ , and the vertical component of velocity  $(v_B)_y$ .

**Vertical Motion.** Since the time of flight and the vertical distance between the ends of the path are known, we can determine  $v_A$ .

$$\begin{aligned} (+\uparrow) \quad y_B &= y_A + (v_A)_y t_{AB} + \frac{1}{2} a_c t_{AB}^2 \\ -1 \text{ m} &= 0 + v_A \sin 30^\circ (1.5 \text{ s}) + \frac{1}{2} (-9.81 \text{ m/s}^2) (1.5 \text{ s})^2 \\ v_A &= 13.38 \text{ m/s} = 13.4 \text{ m/s} \end{aligned} \quad \text{Ans.}$$

**Horizontal Motion.** The range  $R$  can now be determined.

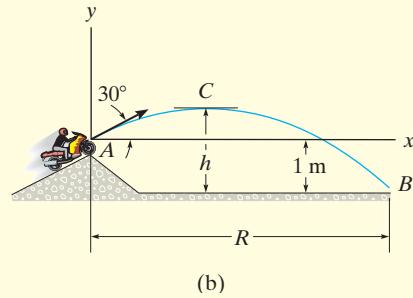
$$\begin{aligned} (\pm) \quad x_B &= x_A + (v_A)_x t_{AB} \\ R &= 0 + 13.38 \cos 30^\circ \text{ m/s} (1.5 \text{ s}) \\ &= 17.4 \text{ m} \end{aligned} \quad \text{Ans.}$$

In order to find the maximum height  $h$  we will consider the path *AC*, Fig. 12–23b. Here the three unknowns are the time of flight  $t_{AC}$ , the horizontal distance from *A* to *C*, and the height  $h$ . At the maximum height  $(v_C)_y = 0$ , and since  $v_A$  is known, we can determine  $h$  directly without considering  $t_{AC}$  using the following equation.

$$\begin{aligned} (v_C)_y^2 &= (v_A)_y^2 + 2a_c[y_C - y_A] \\ 0^2 &= (13.38 \sin 30^\circ \text{ m/s})^2 + 2(-9.81 \text{ m/s}^2)[(h - 1 \text{ m}) - 0] \\ h &= 3.28 \text{ m} \end{aligned} \quad \text{Ans.}$$

**NOTE:** Show that the bike will strike the ground at *B* with a velocity having components of

$$(v_B)_x = 11.6 \text{ m/s} \rightarrow, \quad (v_B)_y = 8.02 \text{ m/s} \downarrow$$

**Fig. 12–23**

## PRELIMINARY PROBLEMS

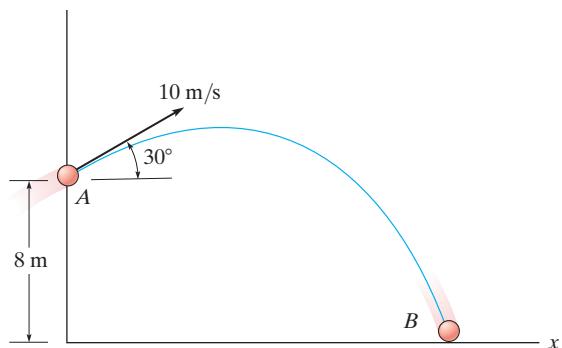
**P12-3.** Use the chain-rule and find  $\dot{y}$  and  $\ddot{y}$  in terms of  $x, \dot{x}$  and  $\ddot{x}$  if

a)  $y = 4x^2$

b)  $y = 3e^x$

c)  $y = 6 \sin x$

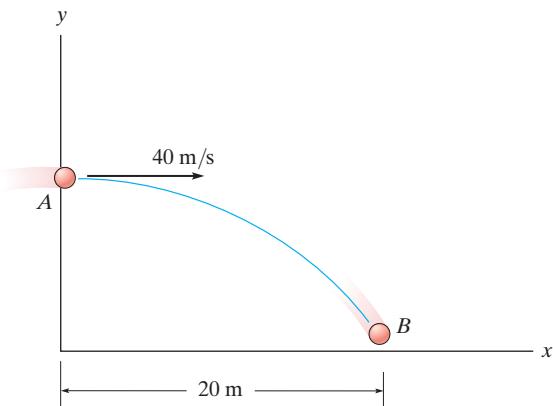
**P12-5.** The particle travels from  $A$  to  $B$ . Identify the three unknowns, and write the three equations needed to solve for them.



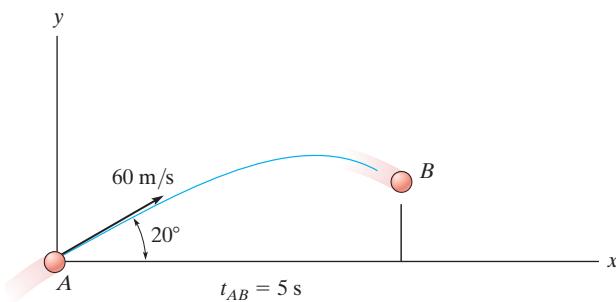
Prob. P12-5

**P12-4.** The particle travels from  $A$  to  $B$ . Identify the three unknowns, and write the three equations needed to solve for them.

**P12-6.** The particle travels from  $A$  to  $B$ . Identify the three unknowns, and write the three equations needed to solve for them.



Prob. P12-4

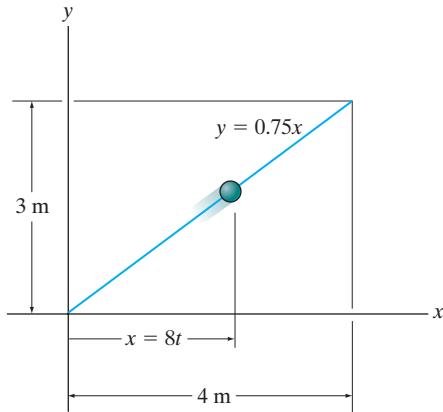


Prob. P12-6

## FUNDAMENTAL PROBLEMS

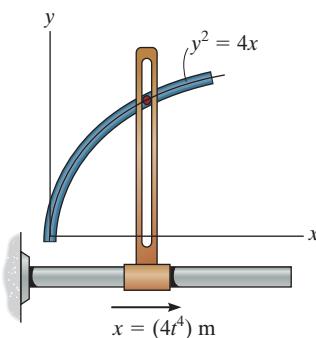
**F12-15.** If the  $x$  and  $y$  components of a particle's velocity are  $v_x = (32t)$  m/s and  $v_y = 8$  m/s, determine the equation of the path  $y = f(x)$ , if  $x = 0$  and  $y = 0$  when  $t = 0$ .

**F12-16.** A particle is traveling along the straight path. If its position along the  $x$  axis is  $x = (8t)$  m, where  $t$  is in seconds, determine its speed when  $t = 2$  s.



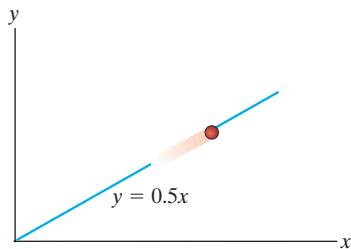
Prob. F12-16

**F12-17.** A particle is constrained to travel along the path. If  $x = (4t^4)$  m, where  $t$  is in seconds, determine the magnitude of the particle's velocity and acceleration when  $t = 0.5$  s.



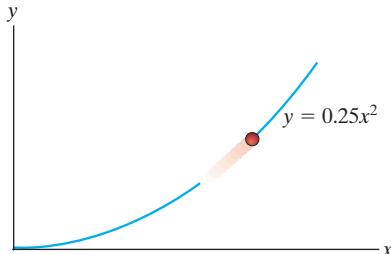
Prob. F12-17

**F12-18.** A particle travels along a straight-line path  $y = 0.5x$ . If the  $x$  component of the particle's velocity is  $v_x = (2t^2)$  m/s, where  $t$  is in seconds, determine the magnitude of the particle's velocity and acceleration when  $t = 4$  s.



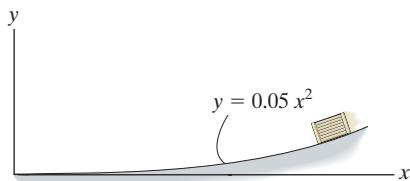
Prob. F12-18

**F12-19.** A particle is traveling along the parabolic path  $y = 0.25x^2$ . If  $x = 8$  m,  $v_x = 8$  m/s, and  $a_x = 4 \text{ m/s}^2$  when  $t = 2$  s, determine the magnitude of the particle's velocity and acceleration at this instant.



Prob. F12-19

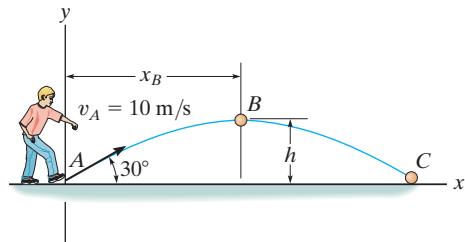
**F12-20.** The box slides down the slope described by the equation  $y = (0.05x^2)$  m, where  $x$  is in meters. If the box has  $x$  components of velocity and acceleration of  $v_x = -3$  m/s and  $a_x = -1.5 \text{ m/s}^2$  at  $x = 5$  m, determine the  $y$  components of the velocity and the acceleration of the box at this instant.



Prob. F12-20

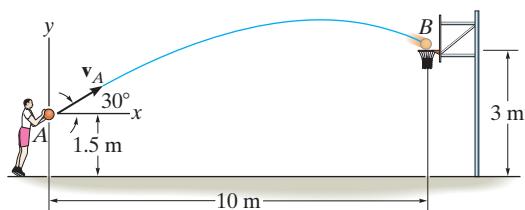
**F12–21.** The ball is kicked from point *A* with the initial velocity  $v_A = 10 \text{ m/s}$ . Determine the maximum height  $h$  it reaches.

**F12–22.** The ball is kicked from point *A* with the initial velocity  $v_A = 10 \text{ m/s}$ . Determine the range  $R$ , and the speed when the ball strikes the ground.



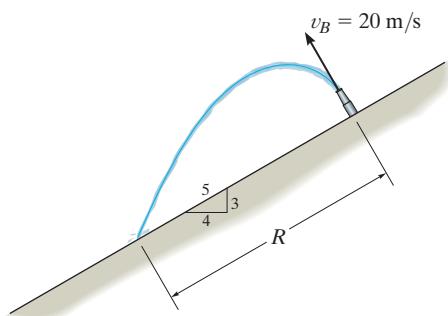
Prob. F12–21/22

**F12–23.** Determine the speed at which the basketball at *A* must be thrown at the angle of  $30^\circ$  so that it makes it to the basket at *B*.



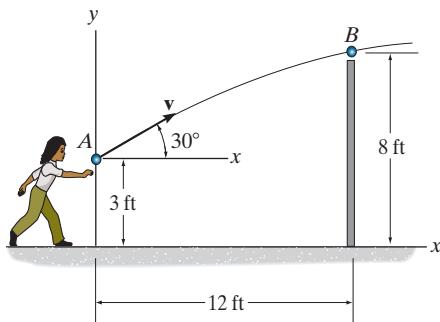
Prob. F12–23

**F12–24.** Water is sprayed at an angle of  $90^\circ$  from the slope at  $20 \text{ m/s}$ . Determine the range  $R$ .



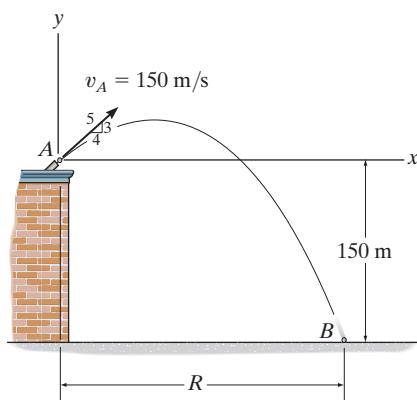
Prob. F12–24

**F12–25.** A ball is thrown from *A*. If it is required to clear the wall at *B*, determine the minimum magnitude of its initial velocity  $v_A$ .



Prob. F12–25

**F12–26.** A projectile is fired with an initial velocity of  $v_A = 150 \text{ m/s}$  off the roof of the building. Determine the range  $R$  where it strikes the ground at *B*.



Prob. F12–26

## PROBLEMS

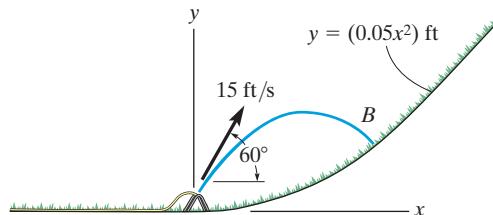
**12-69.** If the velocity of a particle is defined as  $\mathbf{v}(t) = \{0.8t^2\mathbf{i} + 12t^{1/2}\mathbf{j} + 5\mathbf{k}\}$  m/s, determine the magnitude and coordinate direction angles  $\alpha, \beta, \gamma$  of the particle's acceleration when  $t = 2$  s.

**12-70.** The velocity of a particle is  $\mathbf{v} = \{3\mathbf{i} + (6 - 2t)\mathbf{j}\}$  m/s, where  $t$  is in seconds. If  $\mathbf{r} = \mathbf{0}$  when  $t = 0$ , determine the displacement of the particle during the time interval  $t = 1$  s to  $t = 3$  s.

**12-71.** A particle, originally at rest and located at point (3 ft, 2 ft, 5 ft), is subjected to an acceleration of  $\mathbf{a} = \{6t\mathbf{i} + 12t^2\mathbf{k}\}$  ft/s<sup>2</sup>. Determine the particle's position  $(x, y, z)$  at  $t = 1$  s.

\***12-72.** The velocity of a particle is given by  $v = \{16t^2\mathbf{i} + 4t^3\mathbf{j} + (5t + 2)\mathbf{k}\}$  m/s, where  $t$  is in seconds. If the particle is at the origin when  $t = 0$ , determine the magnitude of the particle's acceleration when  $t = 2$  s. Also, what is the  $x, y, z$  coordinate position of the particle at this instant?

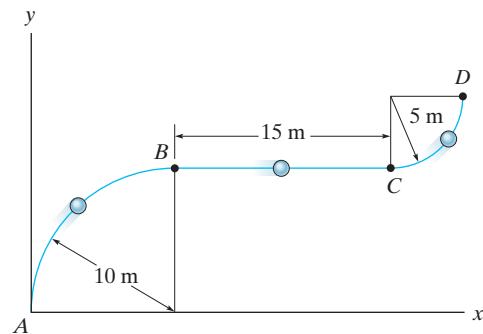
**12-73.** The water sprinkler, positioned at the base of a hill, releases a stream of water with a velocity of 15 ft/s as shown. Determine the point  $B(x, y)$  where the water strikes the ground on the hill. Assume that the hill is defined by the equation  $y = (0.05x^2)$  ft and neglect the size of the sprinkler.



Prob. 12-73

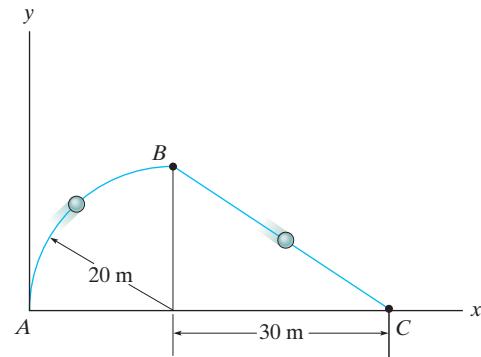
**12-74.** A particle, originally at rest and located at point (3 ft, 2 ft, 5 ft), is subjected to an acceleration  $\mathbf{a} = \{6t\mathbf{i} + 12t^2\mathbf{k}\}$  ft/s<sup>2</sup>. Determine the particle's position  $(x, y, z)$  when  $t = 2$  s.

**12-75.** A particle travels along the curve from  $A$  to  $B$  in 2 s. It takes 4 s for it to go from  $B$  to  $C$  and then 3 s to go from  $C$  to  $D$ . Determine its average speed when it goes from  $A$  to  $D$ .



Prob. 12-75

\***12-76.** A particle travels along the curve from  $A$  to  $B$  in 5 s. It takes 8 s for it to go from  $B$  to  $C$  and then 10 s to go from  $C$  to  $A$ . Determine its average speed when it goes around the closed path.



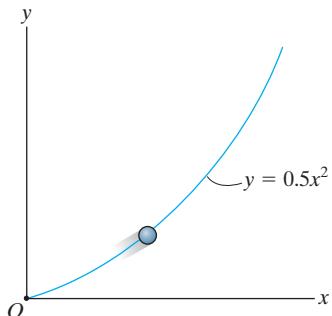
Prob. 12-76

12

**12-77.** The position of a crate sliding down a ramp is given by  $x = (0.25t^3)$  m,  $y = (1.5t^2)$  m,  $z = (6 - 0.75t^{5/2})$  m, where  $t$  is in seconds. Determine the magnitude of the crate's velocity and acceleration when  $t = 2$  s.

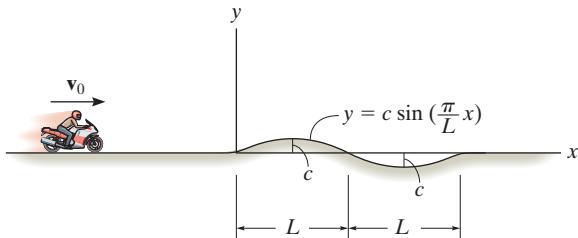
**12-78.** A rocket is fired from rest at  $x = 0$  and travels along a parabolic trajectory described by  $y^2 = [120(10^3)x]$  m. If the  $x$  component of acceleration is  $a_x = \left(\frac{1}{4} t^2\right)$  m/s<sup>2</sup>, where  $t$  is in seconds, determine the magnitude of the rocket's velocity and acceleration when  $t = 10$  s.

**12-79.** The particle travels along the path defined by the parabola  $y = 0.5x^2$ . If the component of velocity along the  $x$  axis is  $v_x = (5t)$  ft/s, where  $t$  is in seconds, determine the particle's distance from the origin  $O$  and the magnitude of its acceleration when  $t = 1$  s. When  $t = 0$ ,  $x = 0$ ,  $y = 0$ .



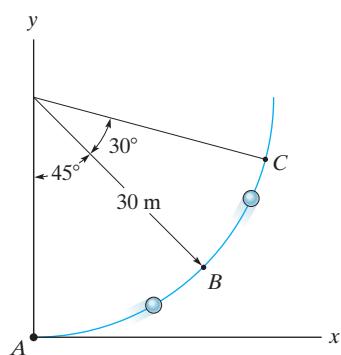
Prob. 12-79

**\*12-80.** The motorcycle travels with constant speed  $v_0$  along the path that, for a short distance, takes the form of a sine curve. Determine the  $x$  and  $y$  components of its velocity at any instant on the curve.



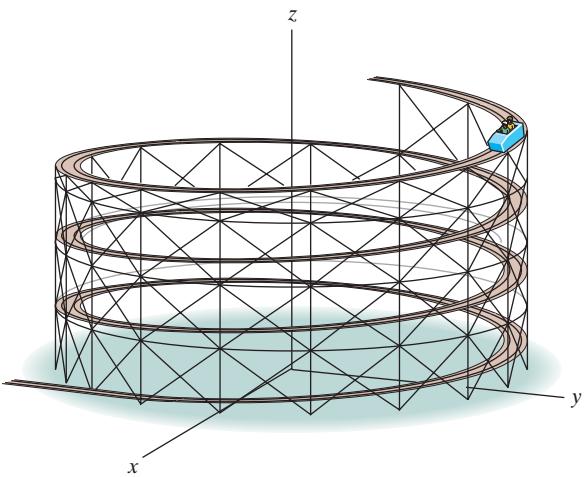
Prob. 12-80

**12-81.** A particle travels along the curve from  $A$  to  $B$  in 1 s. If it takes 3 s for it to go from  $A$  to  $C$ , determine its average velocity when it goes from  $B$  to  $C$ .



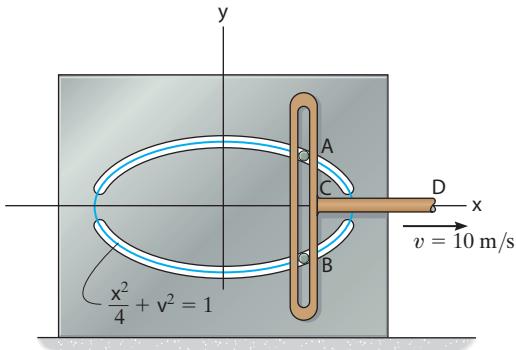
Prob. 12-81

**12-82.** The roller coaster car travels down the helical path at constant speed such that the parametric equations that define its position are  $x = c \sin kt$ ,  $y = c \cos kt$ ,  $z = h - bt$ , where  $c$ ,  $h$ , and  $b$  are constants. Determine the magnitudes of its velocity and acceleration.

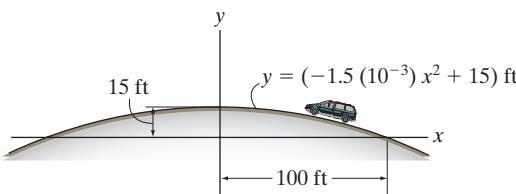


Prob. 12-82

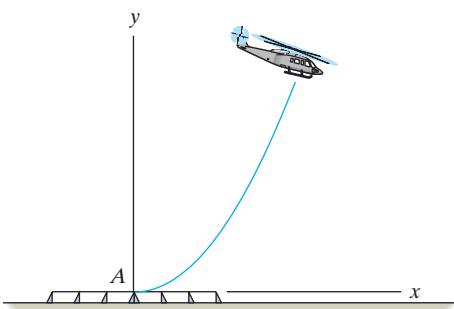
- 12-83.** Pegs A and B are restricted to move in the elliptical slots due to the motion of the slotted link. If the link moves with a constant speed of 10 m/s, determine the magnitude of the velocity and acceleration of peg A when  $x = 1$  m.

**Prob. 12-83**

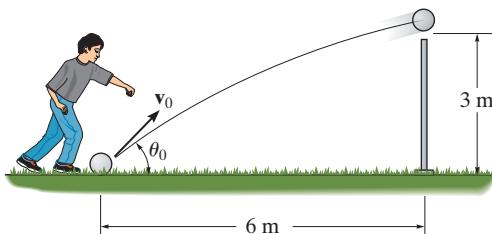
- \*12-84.** The van travels over the hill described by  $y = (-1.5(10^{-3})x^2 + 15)$  ft. If it has a constant speed of 75 ft/s, determine the  $x$  and  $y$  components of the van's velocity and acceleration when  $x = 50$  ft.

**Prob. 12-84**

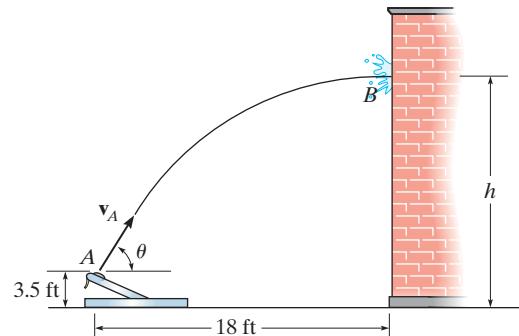
- 12-85.** The flight path of the helicopter as it takes off from A is defined by the parametric equations  $x = (2t^2)$  m and  $y = (0.04t^3)$  m, where  $t$  is the time in seconds. Determine the distance the helicopter is from point A and the magnitudes of its velocity and acceleration when  $t = 10$  s.

**Prob. 12-85**

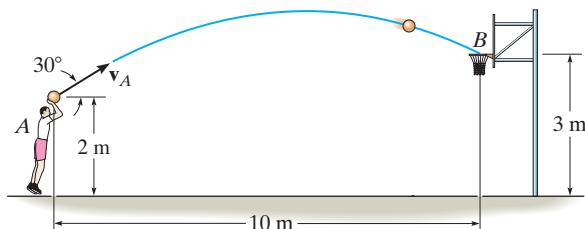
- 12-86.** Determine the minimum initial velocity  $v_0$  and the corresponding angle  $\theta_0$  at which the ball must be kicked in order for it to just cross over the 3-m high fence.

**Prob. 12-86**

- 12-87.** The catapult is used to launch a ball such that it strikes the wall of the building at the maximum height of its trajectory. If it takes 1.5 s to travel from A to B, determine the velocity  $v_A$  at which it was launched, the angle of release  $\theta$ , and the height  $h$ .

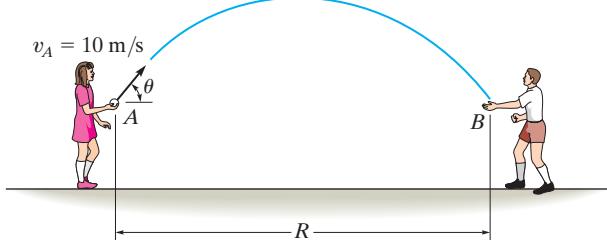
**Prob. 12-87**

- \*12-88.** Neglecting the size of the ball, determine the magnitude  $v_A$  of the basketball's initial velocity and its velocity when it passes through the basket.

**Prob. 12-88**

**12-89.** The girl at  $A$  can throw a ball at  $v_A = 10 \text{ m/s}$ . Calculate the maximum possible range  $R = R_{\max}$  and the associated angle  $\theta$  at which it should be thrown. Assume the ball is caught at  $B$  at the same elevation from which it is thrown.

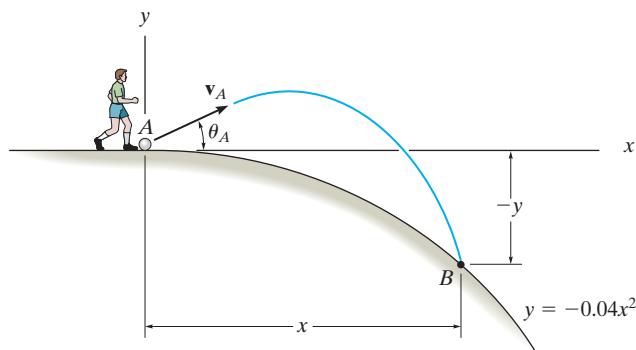
**12-90.** Show that the girl at  $A$  can throw the ball to the boy at  $B$  by launching it at equal angles measured up or down from a  $45^\circ$  inclination. If  $v_A = 10 \text{ m/s}$ , determine the range  $R$  if this value is  $15^\circ$ , i.e.,  $\theta_1 = 45^\circ - 15^\circ = 30^\circ$  and  $\theta_2 = 45^\circ + 15^\circ = 60^\circ$ . Assume the ball is caught at the same elevation from which it is thrown.



Probs. 12-89/90

**12-91.** The ball at  $A$  is kicked with a speed  $v_A = 80 \text{ ft/s}$  and at an angle  $\theta_A = 30^\circ$ . Determine the point  $(x, -y)$  where it strikes the ground. Assume the ground has the shape of a parabola as shown.

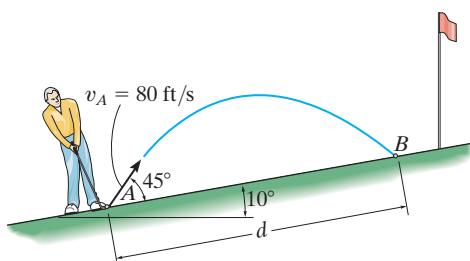
**\*12-92.** The ball at  $A$  is kicked such that  $\theta_A = 30^\circ$ . If it strikes the ground at  $B$  having coordinates  $x = 15 \text{ ft}$ ,  $y = -9 \text{ ft}$ , determine the speed at which it is kicked and the speed at which it strikes the ground.



Probs. 12-91/92

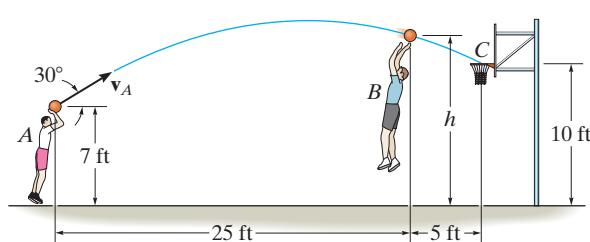
**12-93.** A golf ball is struck with a velocity of  $80 \text{ ft/s}$  as shown. Determine the distance  $d$  to where it will land.

**12-94.** A golf ball is struck with a velocity of  $80 \text{ ft/s}$  as shown. Determine the speed at which it strikes the ground at  $B$  and the time of flight from  $A$  to  $B$ .



Probs. 12-93/94

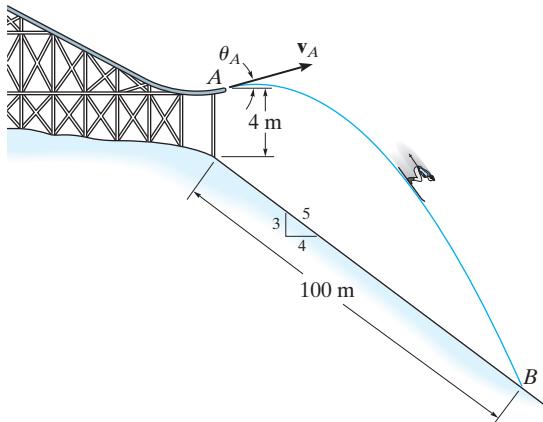
**12-95.** The basketball passed through the hoop even though it barely cleared the hands of the player  $B$  who attempted to block it. Neglecting the size of the ball, determine the magnitude  $v_A$  of its initial velocity and the height  $h$  of the ball when it passes over player  $B$ .



Prob. 12-95

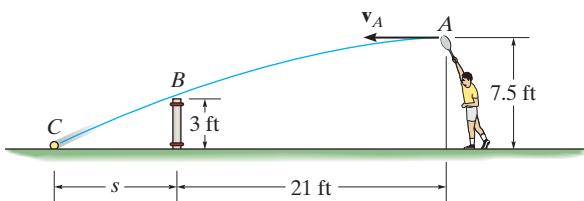
**\*12-96.** It is observed that the skier leaves the ramp  $A$  at an angle  $\theta_A = 25^\circ$  with the horizontal. If he strikes the ground at  $B$ , determine his initial speed  $v_A$  and the time of flight  $t_{AB}$ .

**12-97.** It is observed that the skier leaves the ramp  $A$  at an angle  $\theta_A = 25^\circ$  with the horizontal. If he strikes the ground at  $B$ , determine his initial speed  $v_A$  and the speed at which he strikes the ground.



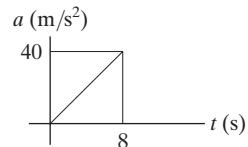
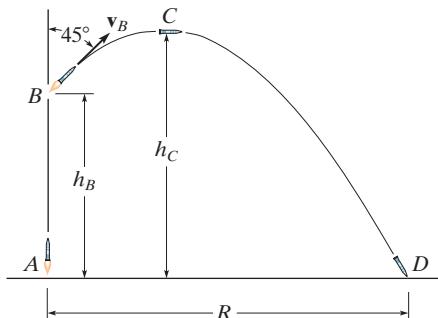
Probs. 12-96/97

**12-98.** Determine the horizontal velocity  $v_A$  of a tennis ball at  $A$  so that it just clears the net at  $B$ . Also, find the distance  $s$  where the ball strikes the ground.



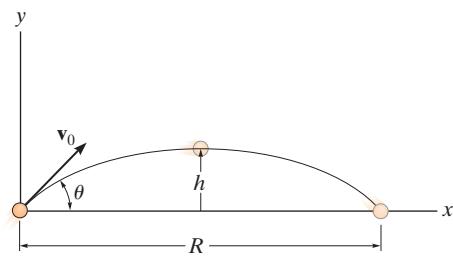
Prob. 12-98

**12-99.** The missile at  $A$  takes off from rest and rises vertically to  $B$ , where its fuel runs out in 8 s. If the acceleration varies with time as shown, determine the missile's height  $h_B$  and speed  $v_B$ . If by internal controls the missile is then suddenly pointed  $45^\circ$  as shown, and allowed to travel in free flight, determine the maximum height attained,  $h_C$ , and the range  $R$  to where it crashes at  $D$ .



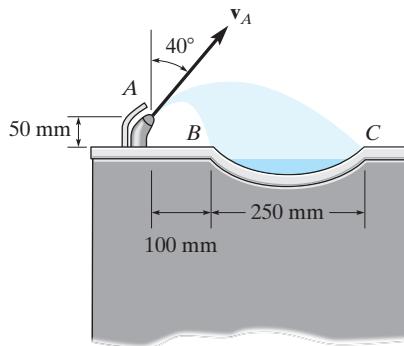
Prob. 12-99

**\*12-100.** The projectile is launched with a velocity  $v_0$ . Determine the range  $R$ , the maximum height  $h$  attained, and the time of flight. Express the results in terms of the angle  $\theta$  and  $v_0$ . The acceleration due to gravity is  $g$ .



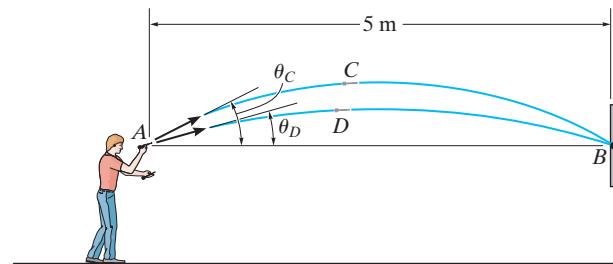
Prob. 12-100

- 12–101.** The drinking fountain is designed such that the nozzle is located from the edge of the basin as shown. Determine the maximum and minimum speed at which water can be ejected from the nozzle so that it does not splash over the sides of the basin at *B* and *C*.



Prob. 12–101

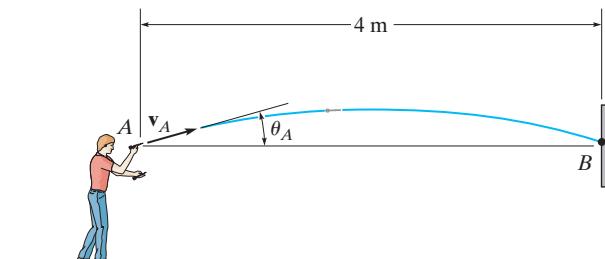
- \*12–104.** The man at *A* wishes to throw two darts at the target at *B* so that they arrive at the *same time*. If each dart is thrown with a speed of 10 m/s, determine the angles  $\theta_C$  and  $\theta_D$  at which they should be thrown and the time between each throw. Note that the first dart must be thrown at  $\theta_C (> \theta_D)$ , then the second dart is thrown at  $\theta_D$ .



Prob. 12–104

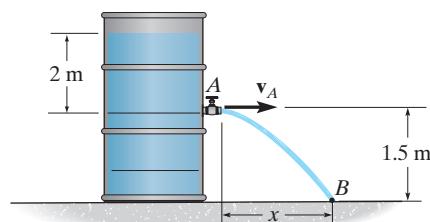
- 12–102.** If the dart is thrown with a speed of 10 m/s, determine the shortest possible time before it strikes the target. Also, what is the corresponding angle  $\theta_A$  at which it should be thrown, and what is the velocity of the dart when it strikes the target?

- 12–103.** If the dart is thrown with a speed of 10 m/s, determine the longest possible time when it strikes the target. Also, what is the corresponding angle  $\theta_A$  at which it should be thrown, and what is the velocity of the dart when it strikes the target?



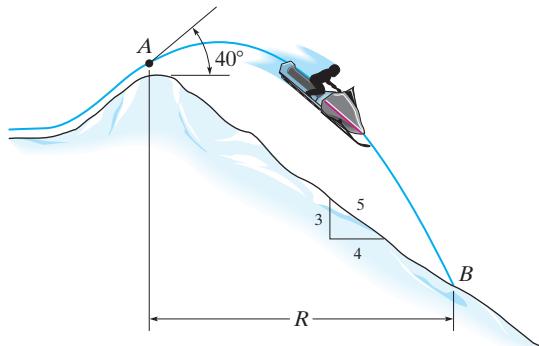
Probs. 12–102/103

- 12–105.** The velocity of the water jet discharging from the orifice can be obtained from  $v = \sqrt{2gh}$ , where  $h = 2$  m is the depth of the orifice from the free water surface. Determine the time for a particle of water leaving the orifice to reach point *B* and the horizontal distance *x* where it hits the surface.



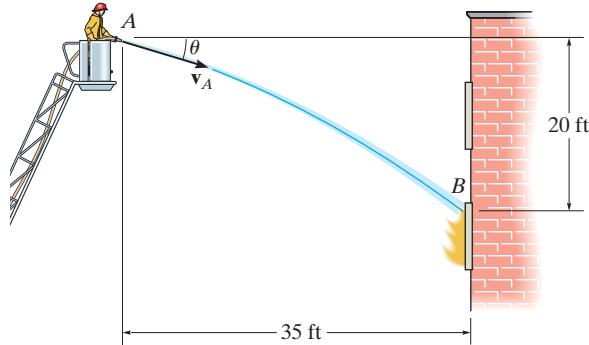
Prob. 12–105

- 12-106.** The snowmobile is traveling at 10 m/s when it leaves the embankment at *A*. Determine the time of flight from *A* to *B* and the range *R* of the trajectory.



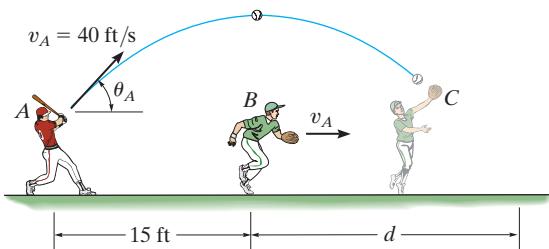
Prob. 12-106

- 12-107.** The fireman wishes to direct the flow of water from his hose to the fire at *B*. Determine two possible angles  $\theta_1$  and  $\theta_2$  at which this can be done. Water flows from the hose at  $v_A = 80 \text{ ft/s}$ .



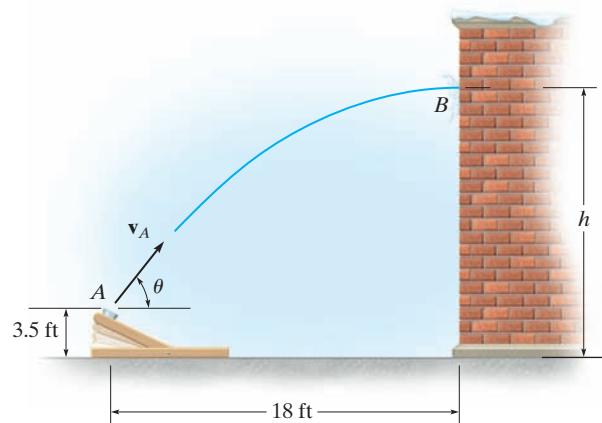
Prob. 12-107

- \*12-108.** The baseball player *A* hits the baseball at  $v_A = 40 \text{ ft/s}$  and  $\theta_A = 60^\circ$  from the horizontal. When the ball is directly overhead of player *B* he begins to run under it. Determine the constant speed at which *B* must run and the distance *d* in order to make the catch at the same elevation at which the ball was hit.



Prob. 12-108

- 12-109.** The catapult is used to launch a ball such that it strikes the wall of the building at the maximum height of its trajectory. If it takes 1.5 s to travel from *A* to *B*, determine the velocity  $\mathbf{v}_A$  at which it was launched, the angle of release  $\theta$ , and the height *h*.



Prob. 12-109

## 12.7 Curvilinear Motion: Normal and Tangential Components

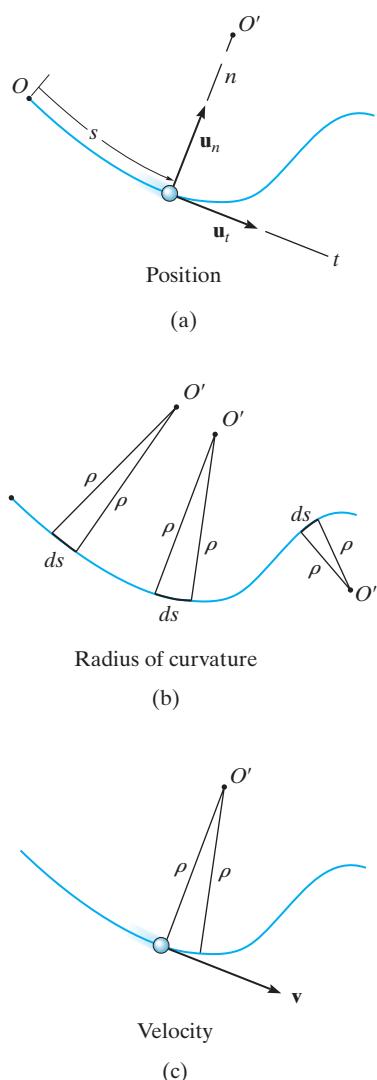


Fig. 12-24

When the path along which a particle travels is *known*, then it is often convenient to describe the motion using  $n$  and  $t$  coordinate axes which act normal and tangent to the path, respectively, and at the instant considered have their *origin located at the particle*.

**Planar Motion.** Consider the particle shown in Fig. 12-24a, which moves in a plane along a fixed curve, such that at a given instant it is at position  $s$ , measured from point  $O$ . We will now consider a coordinate system that has its origin on the curve, and at the instant considered this origin happens to *coincide* with the location of the particle. The  $t$  axis is *tangent* to the curve at the point and is positive in the direction of *increasing  $s$* . We will designate this positive direction with the unit vector  $\mathbf{u}_t$ . A unique choice for the *normal axis* can be made by noting that geometrically the curve is constructed from a series of differential arc segments  $ds$ , Fig. 12-24b. Each segment  $ds$  is formed from the arc of an associated circle having a *radius of curvature*  $\rho$  (rho) and *center of curvature*  $O'$ . The normal axis  $n$  is perpendicular to the  $t$  axis with its positive sense directed *toward* the center of curvature  $O'$ , Fig. 12-24a. This positive direction, which is *always* on the concave side of the curve, will be designated by the unit vector  $\mathbf{u}_n$ . The plane which contains the  $n$  and  $t$  axes is referred to as the *embracing* or *osculating plane*, and in this case it is fixed in the plane of motion.\*

**Velocity.** Since the particle moves,  $s$  is a function of time. As indicated in Sec. 12.4, the particle's velocity  $\mathbf{v}$  has a *direction* that is *always tangent to the path*, Fig. 12-24c, and a *magnitude* that is determined by taking the time derivative of the path function  $s = s(t)$ , i.e.,  $v = ds/dt$  (Eq. 12-8). Hence

$$\mathbf{v} = v \mathbf{u}_t \quad (12-15)$$

where

$$v = \dot{s} \quad (12-16)$$

\*The osculating plane may also be defined as the plane which has the greatest contact with the curve at a point. It is the limiting position of a plane contacting both the point and the arc segment  $ds$ . As noted above, the osculating plane is always coincident with a plane curve; however, each point on a three-dimensional curve has a unique osculating plane.

**Acceleration.** The acceleration of the particle is the time rate of change of the velocity. Thus,

$$\mathbf{a} = \dot{\mathbf{v}} = \dot{v}\mathbf{u}_t + v\dot{\mathbf{u}}_t \quad (12-17)$$

In order to determine the time derivative  $\dot{\mathbf{u}}_t$ , note that as the particle moves along the arc  $ds$  in time  $dt$ ,  $\mathbf{u}_t$  preserves its magnitude of unity; however, its *direction* changes, and becomes  $\mathbf{u}'_t$ , Fig. 12-24d. As shown in Fig. 12-24e, we require  $\mathbf{u}'_t = \mathbf{u}_t + d\mathbf{u}_t$ . Here  $d\mathbf{u}_t$  stretches between the arrowheads of  $\mathbf{u}_t$  and  $\mathbf{u}'_t$ , which lie on an infinitesimal arc of radius  $u_t = 1$ . Hence,  $d\mathbf{u}_t$  has a *magnitude* of  $du_t = (1) d\theta$ , and its *direction* is defined by  $\mathbf{u}_n$ . Consequently,  $d\mathbf{u}_t = d\theta\mathbf{u}_n$ , and therefore the time derivative becomes  $\dot{\mathbf{u}}_t = \dot{\theta}\mathbf{u}_n$ . Since  $ds = \rho d\theta$ , Fig. 12-24d, then  $\dot{\theta} = \dot{s}/\rho$ , and therefore

$$\dot{\mathbf{u}}_t = \dot{\theta}\mathbf{u}_n = \frac{\dot{s}}{\rho}\mathbf{u}_n = \frac{v}{\rho}\mathbf{u}_n$$

Substituting into Eq. 12-17,  $\mathbf{a}$  can be written as the sum of its two components,

$$\mathbf{a} = a_t\mathbf{u}_t + a_n\mathbf{u}_n \quad (12-18)$$

where

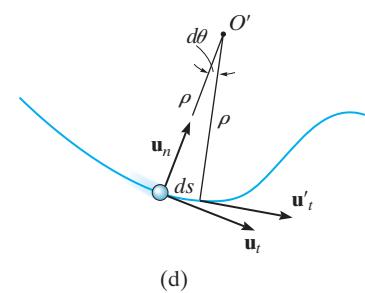
$$a_t = \dot{v} \quad \text{or} \quad a_t ds = v dv \quad (12-19)$$

and

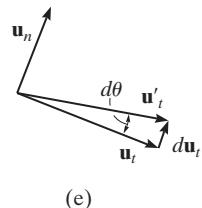
$$a_n = \frac{v^2}{\rho} \quad (12-20)$$

These two mutually perpendicular components are shown in Fig. 12-24f. Therefore, the *magnitude* of acceleration is the positive value of

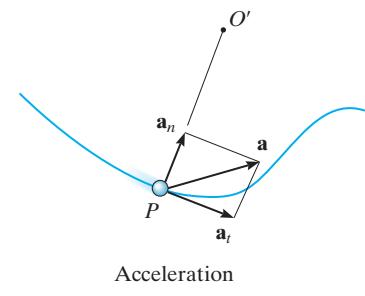
$$a = \sqrt{a_t^2 + a_n^2} \quad (12-21)$$



(d)



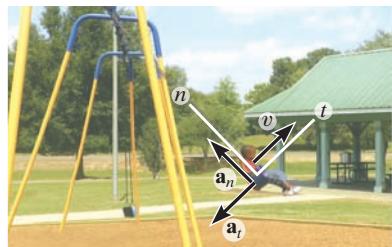
(e)



Acceleration

(f)

**Fig. 12-24 (cont.)**



As the boy swings upward with a velocity  $\mathbf{v}$ , his motion can be analyzed using  $n$ - $t$  coordinates. As he rises, the magnitude of his velocity (speed) is decreasing, and so  $a_t$  will be negative. The rate at which the direction of his velocity changes is  $a_n$ , which is always positive, that is, towards the center of rotation. (© R.C. Hibbeler)

To better understand these results, consider the following two special cases of motion.

1. If the particle moves along a straight line, then  $\rho \rightarrow \infty$  and from Eq. 12-20,  $a_n = 0$ . Thus  $\mathbf{a} = \mathbf{a}_t = \dot{\mathbf{v}}$ , and we can conclude that the *tangential component of acceleration represents the time rate of change in the magnitude of the velocity*.
2. If the particle moves along a curve with a constant speed, then  $a_t = \dot{v} = 0$  and  $\mathbf{a} = \mathbf{a}_n = v^2/\rho$ . Therefore, the *normal component of acceleration represents the time rate of change in the direction of the velocity*. Since  $\mathbf{a}_n$  always acts towards the center of curvature, this component is sometimes referred to as the *centripetal* (or center seeking) *acceleration*.

As a result of these interpretations, a particle moving along the curved path in Fig. 12-25 will have accelerations directed as shown.

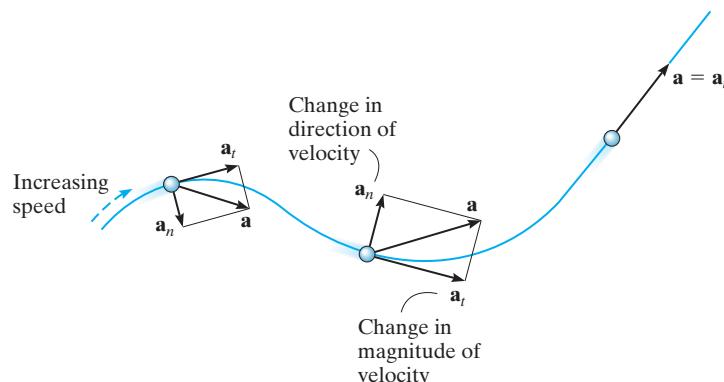


Fig. 12-25

**Three-Dimensional Motion.** If the particle moves along a space curve, Fig. 12-26, then at a given instant the  $t$  axis is uniquely specified; however, an infinite number of straight lines can be constructed normal to the tangent axis. As in the case of planar motion, we will choose the positive  $n$  axis directed toward the path's center of curvature  $O'$ . This axis is referred to as the *principal normal* to the curve. With the  $n$  and  $t$  axes so defined, Eqs. 12-15 through 12-21 can be used to determine  $\mathbf{v}$  and  $\mathbf{a}$ . Since  $\mathbf{u}_t$  and  $\mathbf{u}_n$  are always perpendicular to one another and lie in the osculating plane, for spatial motion a third unit vector,  $\mathbf{u}_b$ , defines the *binormal axis*  $b$  which is perpendicular to  $\mathbf{u}_t$  and  $\mathbf{u}_n$ , Fig. 12-26.

Since the three unit vectors are related to one another by the vector cross product, e.g.,  $\mathbf{u}_b = \mathbf{u}_t \times \mathbf{u}_n$ , Fig. 12-26, it may be possible to use this relation to establish the direction of one of the axes, if the directions of the other two are known. For example, no motion occurs in the  $\mathbf{u}_b$  direction, and if this direction and  $\mathbf{u}_t$  are known, then  $\mathbf{u}_n$  can be determined, where in this case  $\mathbf{u}_n = \mathbf{u}_b \times \mathbf{u}_t$ , Fig. 12-26. Remember, though, that  $\mathbf{u}_n$  is always on the concave side of the curve.

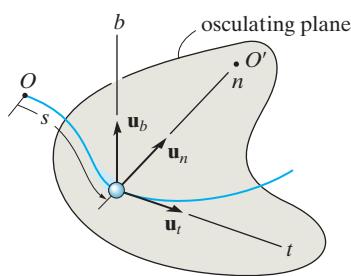


Fig. 12-26

## Procedure for Analysis

### Coordinate System.

- Provided the *path* of the particle is *known*, we can establish a set of  $n$  and  $t$  coordinates having a *fixed origin*, which is coincident with the particle at the instant considered.
- The positive tangent axis acts in the direction of motion and the positive normal axis is directed toward the path's center of curvature.

### Velocity.

- The particle's *velocity* is always tangent to the path.
- The magnitude of velocity is found from the time derivative of the path function.

$$v = \dot{s}$$

### Tangential Acceleration.

- The tangential component of acceleration is the result of the time rate of change in the *magnitude* of velocity. This component acts in the positive  $s$  direction if the particle's speed is increasing or in the opposite direction if the speed is decreasing.
- The relations between  $a_t$ ,  $v$ ,  $t$ , and  $s$  are the same as for rectilinear motion, namely,

$$a_t = \dot{v} \quad a_t ds = v dv$$

- If  $a_t$  is constant,  $a_t = (a_t)_c$ , the above equations, when integrated, yield

$$\begin{aligned}s &= s_0 + v_0 t + \frac{1}{2}(a_t)_c t^2 \\v &= v_0 + (a_t)_c t \\v^2 &= v_0^2 + 2(a_t)_c(s - s_0)\end{aligned}$$

### Normal Acceleration.

- The normal component of acceleration is the result of the time rate of change in the *direction* of the velocity. This component is *always* directed toward the center of curvature of the path, i.e., along the positive  $n$  axis.
- The magnitude of this component is determined from

$$a_n = \frac{v^2}{\rho}$$

- If the path is expressed as  $y = f(x)$ , the radius of curvature  $\rho$  at any point on the path is determined from the equation

$$\rho = \frac{[1 + (dy/dx)^2]^{3/2}}{|d^2y/dx^2|}$$

The derivation of this result is given in any standard calculus text.



Once the rotation is constant, the riders will then have only a normal component of acceleration. (© R.C. Hibbeler)



Motorists traveling along this cloverleaf interchange experience a normal acceleration due to the change in direction of their velocity. A tangential component of acceleration occurs when the cars' speed is increased or decreased. (© R.C. Hibbeler)

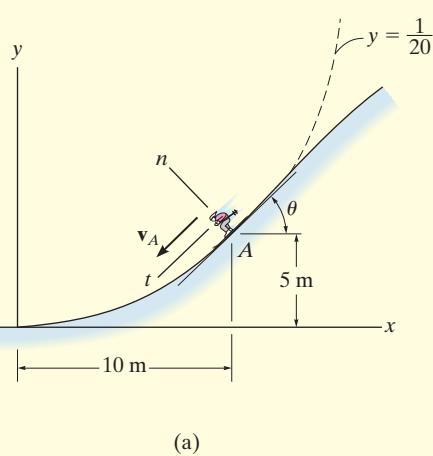
## EXAMPLE | 12.14

When the skier reaches point *A* along the parabolic path in Fig. 12-27a, he has a speed of 6 m/s which is increasing at 2 m/s<sup>2</sup>. Determine the direction of his velocity and the direction and magnitude of his acceleration at this instant. Neglect the size of the skier in the calculation.

## SOLUTION

**Coordinate System.** Although the path has been expressed in terms of its *x* and *y* coordinates, we can still establish the origin of the *n*, *t* axes at the fixed point *A* on the path and determine the components of  $\mathbf{v}$  and  $\mathbf{a}$  along these axes, Fig. 12-27a.

**Velocity.** By definition, the velocity is always directed tangent to the path. Since  $y = \frac{1}{20}x^2$ ,  $dy/dx = \frac{1}{10}x$ , then at  $x = 10$  m,  $dy/dx = 1$ . Hence, at *A*,  $\mathbf{v}$  makes an angle of  $\theta = \tan^{-1} 1 = 45^\circ$  with the *x* axis, Fig. 12-27b. Therefore,



(a)

$$v_A = 6 \text{ m/s } 45^\circ \swarrow \quad \text{Ans.}$$

The acceleration is determined from  $\mathbf{a} = \dot{v}\mathbf{u}_t + (v^2/\rho)\mathbf{u}_n$ . However, it is first necessary to determine the radius of curvature of the path at *A* (10 m, 5 m). Since  $d^2y/dx^2 = \frac{1}{10}$ , then

$$\rho = \frac{[1 + (dy/dx)^2]^{3/2}}{|d^2y/dx^2|} = \frac{\left[1 + \left(\frac{1}{10}x\right)^2\right]^{3/2}}{\left|\frac{1}{10}\right|} \Big|_{x=10 \text{ m}} = 28.28 \text{ m}$$

The acceleration becomes

$$\begin{aligned} \mathbf{a}_A &= \dot{v}\mathbf{u}_t + \frac{v^2}{\rho}\mathbf{u}_n \\ &= 2\mathbf{u}_t + \frac{(6 \text{ m/s})^2}{28.28 \text{ m}}\mathbf{u}_n \\ &= \{2\mathbf{u}_t + 1.273\mathbf{u}_n\} \text{ m/s}^2 \end{aligned}$$

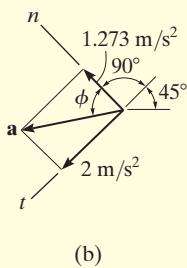
As shown in Fig. 12-27b,

$$a = \sqrt{(2 \text{ m/s}^2)^2 + (1.273 \text{ m/s}^2)^2} = 2.37 \text{ m/s}^2$$

$$\phi = \tan^{-1} \frac{2}{1.273} = 57.5^\circ$$

Thus,  $45^\circ + 90^\circ + 57.5^\circ - 180^\circ = 12.5^\circ$  so that,

$$a = 2.37 \text{ m/s}^2 \ 12.5^\circ \swarrow \quad \text{Ans.}$$



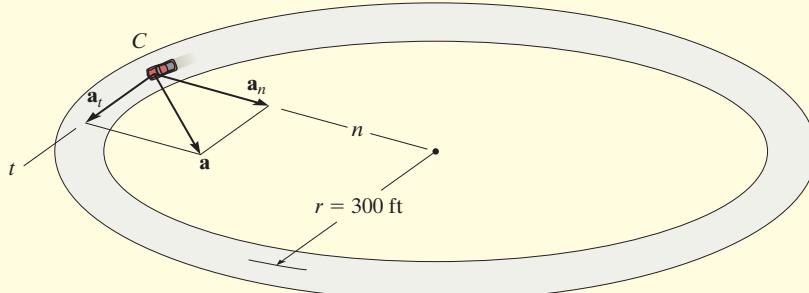
(b)

Fig. 12-27

**NOTE:** By using *n*, *t* coordinates, we were able to readily solve this problem through the use of Eq. 12-18, since it accounts for the separate changes in the magnitude and direction of  $\mathbf{v}$ .

**EXAMPLE | 12.15**

A race car *C* travels around the horizontal circular track that has a radius of 300 ft, Fig. 12–28. If the car increases its speed at a constant rate of  $7 \text{ ft/s}^2$ , starting from rest, determine the time needed for it to reach an acceleration of  $8 \text{ ft/s}^2$ . What is its speed at this instant?

**Fig. 12–28****SOLUTION**

**Coordinate System.** The origin of the *n* and *t* axes is coincident with the car at the instant considered. The *t* axis is in the direction of motion, and the positive *n* axis is directed toward the center of the circle. This coordinate system is selected since the path is known.

**Acceleration.** The magnitude of acceleration can be related to its components using  $a = \sqrt{a_t^2 + a_n^2}$ . Here  $a_t = 7 \text{ ft/s}^2$ . Since  $a_n = v^2/\rho$ , the velocity as a function of time must be determined first.

$$\begin{aligned} v &= v_0 + (a_t)_c t \\ v &= 0 + 7t \end{aligned}$$

Thus

$$a_n = \frac{v^2}{\rho} = \frac{(7t)^2}{300} = 0.163t^2 \text{ ft/s}^2$$

The time needed for the acceleration to reach  $8 \text{ ft/s}^2$  is therefore

$$\begin{aligned} a &= \sqrt{a_t^2 + a_n^2} \\ 8 \text{ ft/s}^2 &= \sqrt{(7 \text{ ft/s}^2)^2 + (0.163t^2)^2} \end{aligned}$$

Solving for the positive value of *t* yields

$$\begin{aligned} 0.163t^2 &= \sqrt{(8 \text{ ft/s}^2)^2 - (7 \text{ ft/s}^2)^2} \\ t &= 4.87 \text{ s} \quad \text{Ans.} \end{aligned}$$

**Velocity.** The speed at time *t* = 4.87 s is

$$v = 7t = 7(4.87) = 34.1 \text{ ft/s} \quad \text{Ans.}$$

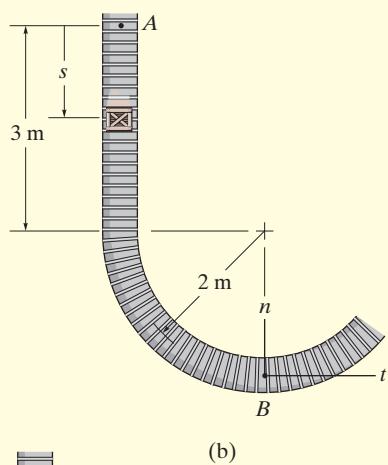
**NOTE:** Remember the velocity will always be tangent to the path, whereas the acceleration will be directed within the curvature of the path.

## EXAMPLE | 12.16



(© R.C. Hibbeler)

(a)



(b)

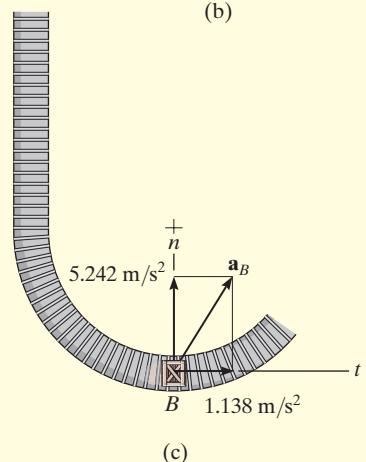


Fig. 12-29

The boxes in Fig. 12-29a travel along the industrial conveyor. If a box as in Fig. 12-29b starts from rest at *A* and increases its speed such that  $a_t = (0.2t) \text{ m/s}^2$ , where *t* is in seconds, determine the magnitude of its acceleration when it arrives at point *B*.

## SOLUTION

**Coordinate System.** The position of the box at any instant is defined from the fixed point *A* using the position or path coordinate *s*, Fig. 12-29b. The acceleration is to be determined at *B*, so the origin of the *n, t* axes is at this point.

**Acceleration.** To determine the acceleration components  $a_t = \dot{v}$  and  $a_n = v^2/\rho$ , it is first necessary to formulate *v* and  $\dot{v}$  so that they may be evaluated at *B*. Since  $v_A = 0$  when *t* = 0, then

$$a_t = \dot{v} = 0.2t \quad (1)$$

$$\int_0^v dv = \int_0^t 0.2t dt$$

$$v = 0.1t^2 \quad (2)$$

The time needed for the box to reach point *B* can be determined by realizing that the position of *B* is  $s_B = 3 + 2\pi(2)/4 = 6.142 \text{ m}$ , Fig. 12-29b, and since  $s_A = 0$  when *t* = 0 we have

$$v = \frac{ds}{dt} = 0.1t^2$$

$$\int_0^{6.142 \text{ m}} ds = \int_0^{t_B} 0.1t^2 dt$$

$$6.142 \text{ m} = 0.0333t_B^3$$

$$t_B = 5.690 \text{ s}$$

Substituting into Eqs. 1 and 2 yields

$$(a_B)_t = \dot{v}_B = 0.2(5.690) = 1.138 \text{ m/s}^2$$

$$v_B = 0.1(5.69)^2 = 3.238 \text{ m/s}$$

At *B*,  $\rho_B = 2 \text{ m}$ , so that

$$(a_B)_n = \frac{v_B^2}{\rho_B} = \frac{(3.238 \text{ m/s})^2}{2 \text{ m}} = 5.242 \text{ m/s}^2$$

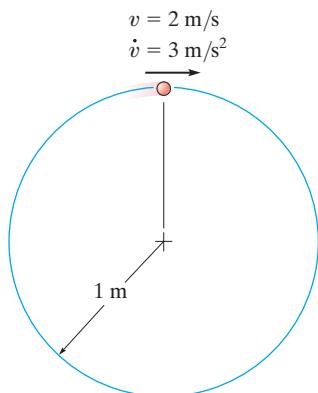
The magnitude of  $\mathbf{a}_B$ , Fig. 12-29c, is therefore

$$a_B = \sqrt{(1.138 \text{ m/s}^2)^2 + (5.242 \text{ m/s}^2)^2} = 5.36 \text{ m/s}^2 \quad \text{Ans.}$$

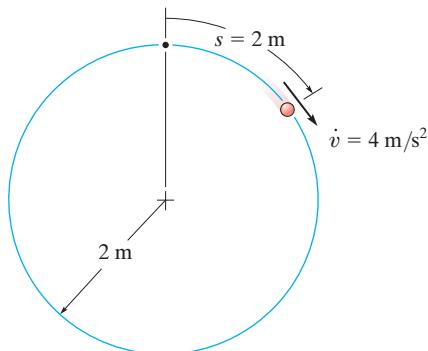
## PRELIMINARY PROBLEM

**P12-7.**

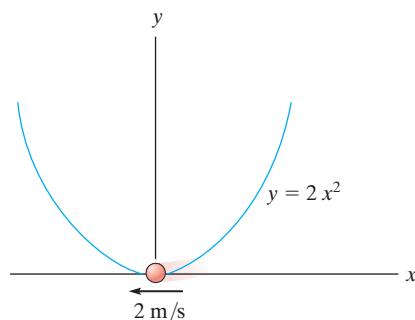
- a) Determine the acceleration at the instant shown.



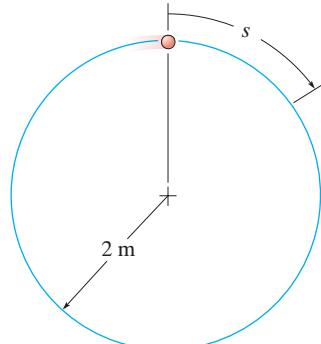
- b) Determine the increase in speed and the normal component of acceleration at  $s = 2 \text{ m}$ . At  $s = 0$ ,  $v = 0$ .



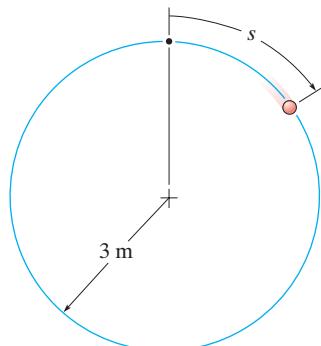
- c) Determine the acceleration at the instant shown. The particle has a constant speed of  $2 \text{ m/s}$ .



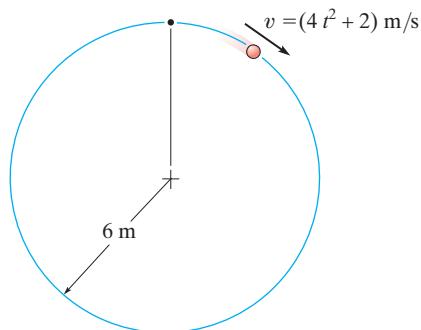
- d) Determine the normal and tangential components of acceleration at  $s = 0$  if  $v = (4s + 1) \text{ m/s}$ , where  $s$  is in meters.



- e) Determine the acceleration at  $s = 2 \text{ m}$  if  $\dot{v} = (2s) \text{ m/s}^2$ , where  $s$  is in meters. At  $s = 0$ ,  $v = 1 \text{ m/s}$ .



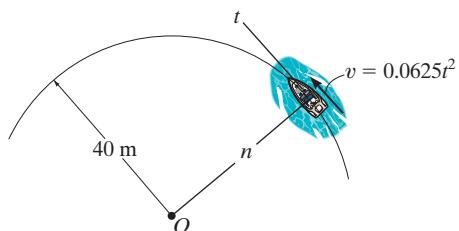
- f. Determine the acceleration when  $t = 1 \text{ s}$  if  $v = (4t^2 + 2) \text{ m/s}$ , where  $t$  is in seconds.



**Prob. P12-7**

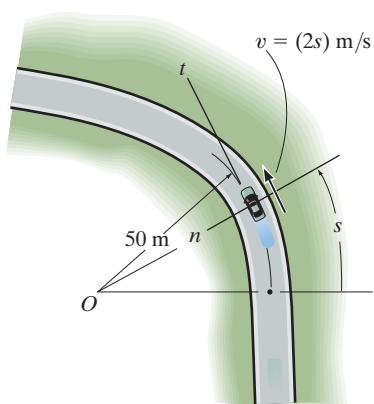
## FUNDAMENTAL PROBLEMS

**F12–27.** The boat is traveling along the circular path with a speed of  $v = (0.0625t^2)$  m/s, where  $t$  is in seconds. Determine the magnitude of its acceleration when  $t = 10$  s.



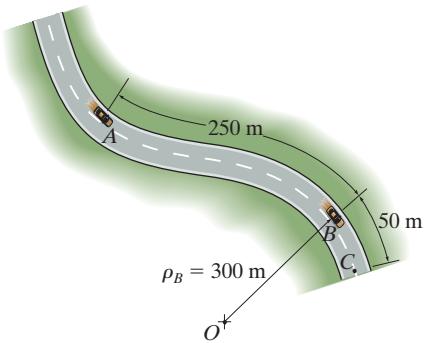
Prob. F12-27

**F12–28.** The car is traveling along the road with a speed of  $v = (2s)$  m/s, where  $s$  is in meters. Determine the magnitude of its acceleration when  $s = 10$  m.



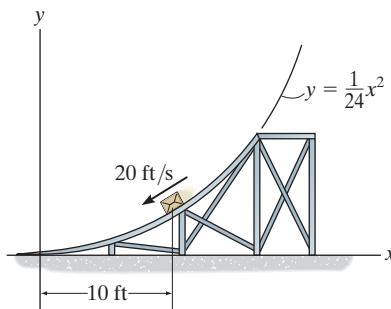
Prob. F12-28

**F12–29.** If the car decelerates uniformly along the curved road from 25 m/s at  $A$  to 15 m/s at  $C$ , determine the acceleration of the car at  $B$ .



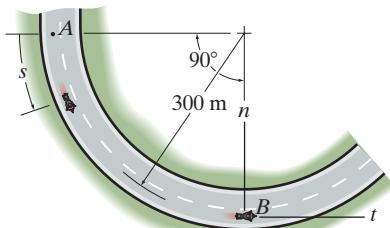
Prob. F12-29

**F12–30.** When  $x = 10$  ft, the crate has a speed of 20 ft/s which is increasing at 6 ft/s<sup>2</sup>. Determine the direction of the crate's velocity and the magnitude of the crate's acceleration at this instant.



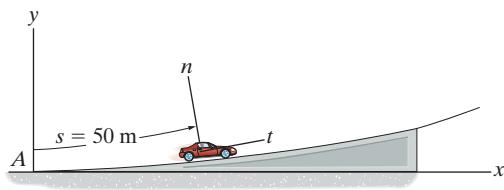
Prob. F12-30

**F12–31.** If the motorcycle has a deceleration of  $a_t = -(0.001s)$  m/s<sup>2</sup> and its speed at position  $A$  is 25 m/s, determine the magnitude of its acceleration when it passes point  $B$ .



Prob. F12-31

**F12–32.** The car travels up the hill with a speed of  $v = (0.2s)$  m/s, where  $s$  is in meters, measured from  $A$ . Determine the magnitude of its acceleration when it is at point  $s = 50$  m, where  $\rho = 500$  m.



Prob. F12-32

## PROBLEMS

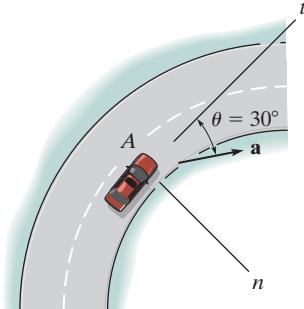
**12-110.** An automobile is traveling on a curve having a radius of 800 ft. If the acceleration of the automobile is  $5 \text{ ft/s}^2$ , determine the constant speed at which the automobile is traveling.

**12-111.** Determine the maximum constant speed a race car can have if the acceleration of the car cannot exceed  $7.5 \text{ m/s}^2$  while rounding a track having a radius of curvature of 200 m.

**\*12-112.** A boat has an initial speed of 16 ft/s. If it then increases its speed along a circular path of radius  $\rho = 80 \text{ ft}$  at the rate of  $\dot{v} = (1.5s) \text{ ft/s}$ , where  $s$  is in feet, determine the time needed for the boat to travel  $s = 50 \text{ ft}$ .

**12-113.** The position of a particle is defined by  $\mathbf{r} = \{4(t - \sin t)\mathbf{i} + (2t^2 - 3)\mathbf{j}\} \text{ m}$ , where  $t$  is in seconds and the argument for the sine is in radians. Determine the speed of the particle and its normal and tangential components of acceleration when  $t = 1 \text{ s}$ .

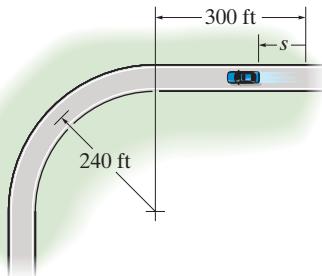
**12-114.** The automobile has a speed of 80 ft/s at point  $A$  and an acceleration having a magnitude of  $10 \text{ ft/s}^2$ , acting in the direction shown. Determine the radius of curvature of the path at point  $A$  and the tangential component of acceleration.



Prob. 12-114

**12-115.** The automobile is originally at rest at  $s = 0$ . If its speed is increased by  $\dot{v} = (0.05t^2) \text{ ft/s}^2$ , where  $t$  is in seconds, determine the magnitudes of its velocity and acceleration when  $t = 18 \text{ s}$ .

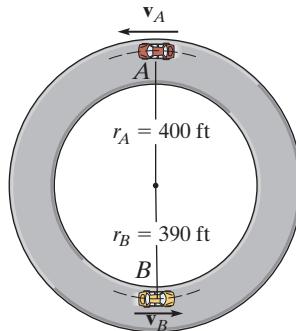
**\*12-116.** The automobile is originally at rest  $s = 0$ . If it then starts to increase its speed at  $\dot{v} = (0.05t^2) \text{ ft/s}^2$ , where  $t$  is in seconds, determine the magnitudes of its velocity and acceleration at  $s = 550 \text{ ft}$ .



Probs. 12-115/116

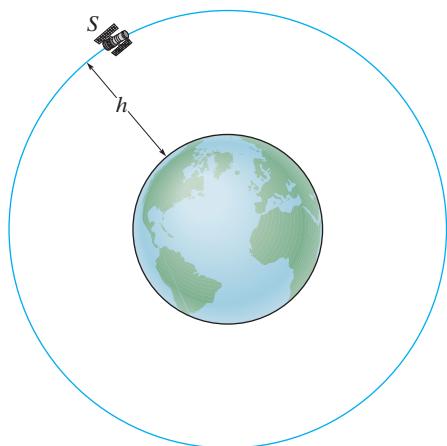
**12-117.** The two cars  $A$  and  $B$  travel along the circular path at constant speeds  $v_A = 80 \text{ ft/s}$  and  $v_B = 100 \text{ ft/s}$ , respectively. If they are at the positions shown when  $t = 0$ , determine the time when the cars are side by side, and the time when they are  $90^\circ$  apart.

**12-118.** Cars  $A$  and  $B$  are traveling around the circular race track. At the instant shown,  $A$  has a speed of 60 ft/s and is increasing its speed at the rate of  $15 \text{ ft/s}^2$  until it travels for a distance of  $100\pi \text{ ft}$ , after which it maintains a constant speed. Car  $B$  has a speed of 120 ft/s and is decreasing its speed at  $15 \text{ ft/s}^2$  until it travels a distance of  $65\pi \text{ ft}$ , after which it maintains a constant speed. Determine the time when they come side by side.



Probs. 12-117/118

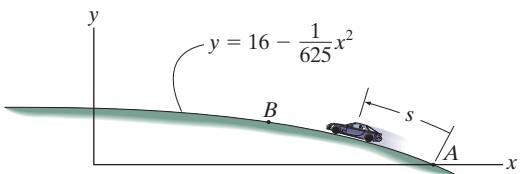
- 12-119.** The satellite  $S$  travels around the earth in a circular path with a constant speed of 20 Mm/h. If the acceleration is  $2.5 \text{ m/s}^2$ , determine the altitude  $h$ . Assume the earth's diameter to be 12 713 km.



Prob. 12-119

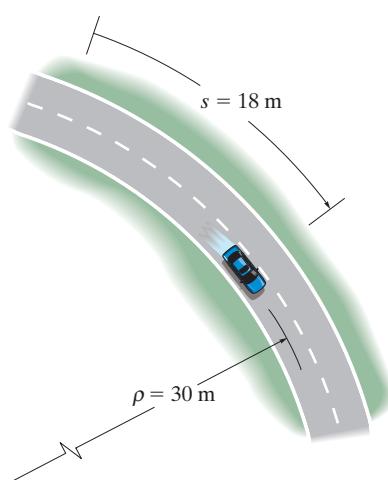
- 12-121.** The car passes point  $A$  with a speed of 25 m/s after which its speed is defined by  $v = (25 - 0.15s) \text{ m/s}$ . Determine the magnitude of the car's acceleration when it reaches point  $B$ , where  $s = 51.5 \text{ m}$  and  $x = 50 \text{ m}$ .

- 12-122.** If the car passes point  $A$  with a speed of 20 m/s and begins to increase its speed at a constant rate of  $a_t = 0.5 \text{ m/s}^2$ , determine the magnitude of the car's acceleration when  $s = 101.68 \text{ m}$  and  $x = 0$ .



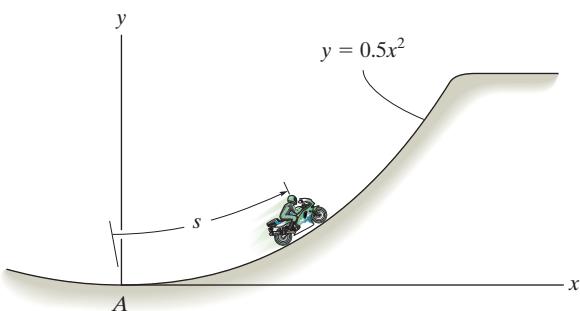
Probs. 12-121/122

- \*12-120.** The car travels along the circular path such that its speed is increased by  $a_t = (0.5e^t) \text{ m/s}^2$ , where  $t$  is in seconds. Determine the magnitudes of its velocity and acceleration after the car has traveled  $s = 18 \text{ m}$  starting from rest. Neglect the size of the car.



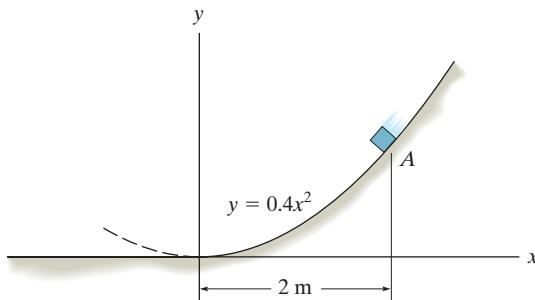
Prob. 12-120

- 12-123.** The motorcycle is traveling at 1 m/s when it is at  $A$ . If the speed is then increased at  $\dot{v} = 0.1 \text{ m/s}^2$ , determine its speed and acceleration at the instant  $t = 5 \text{ s}$ .



Prob. 12-123

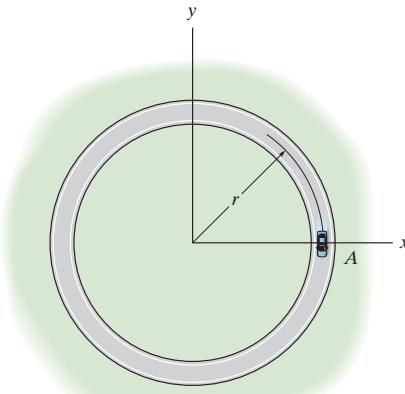
- \*12-124.** The box of negligible size is sliding down along a curved path defined by the parabola  $y = 0.4x^2$ . When it is at  $A(x_A = 2 \text{ m}, y_A = 1.6 \text{ m})$ , the speed is  $v = 8 \text{ m/s}$  and the increase in speed is  $dv/dt = 4 \text{ m/s}^2$ . Determine the magnitude of the acceleration of the box at this instant.



Prob. 12-124

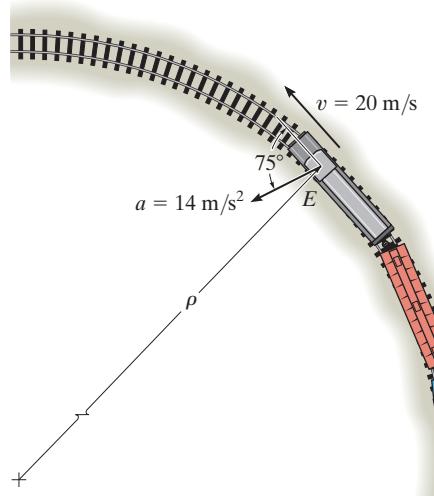
- 12-125.** The car travels around the circular track having a radius of  $r = 300 \text{ m}$  such that when it is at point  $A$  it has a velocity of  $5 \text{ m/s}$ , which is increasing at the rate of  $\dot{v} = (0.06t) \text{ m/s}^2$ , where  $t$  is in seconds. Determine the magnitudes of its velocity and acceleration when it has traveled one-third the way around the track.

- 12-126.** The car travels around the portion of a circular track having a radius of  $r = 500 \text{ ft}$  such that when it is at point  $A$  it has a velocity of  $2 \text{ ft/s}$ , which is increasing at the rate of  $\dot{v} = (0.002t) \text{ ft/s}^2$ , where  $t$  is in seconds. Determine the magnitudes of its velocity and acceleration when it has traveled three-fourths the way around the track.



Probs. 12-125/126

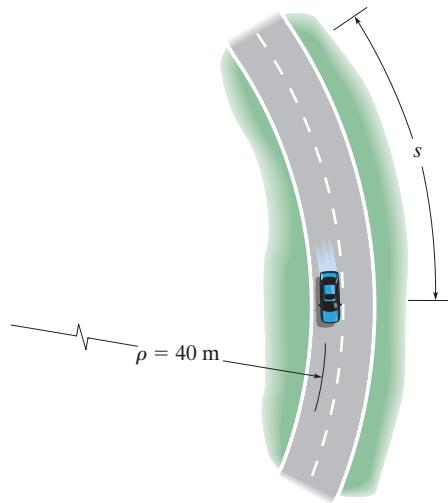
- 12-127.** At a given instant the train engine at  $E$  has a speed of  $20 \text{ m/s}$  and an acceleration of  $14 \text{ m/s}^2$  acting in the direction shown. Determine the rate of increase in the train's speed and the radius of curvature  $\rho$  of the path.



Prob. 12-127

- \*12-128.** The car has an initial speed  $v_0 = 20 \text{ m/s}$ . If it increases its speed along the circular track at  $s = 0$ ,  $a_t = (0.8s) \text{ m/s}^2$ , where  $s$  is in meters, determine the time needed for the car to travel  $s = 25 \text{ m}$ .

- 12-129.** The car starts from rest at  $s = 0$  and increases its speed at  $a_t = 4 \text{ m/s}^2$ . Determine the time when the magnitude of acceleration becomes  $20 \text{ m/s}^2$ . At what position  $s$  does this occur?



Probs. 12-128/129

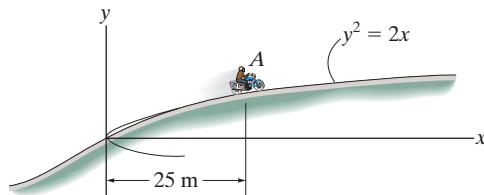
**12-130.** A boat is traveling along a circular curve having a radius of 100 ft. If its speed at  $t = 0$  is 15 ft/s and is increasing at  $\dot{v} = (0.8t)$  ft/s $^2$ , determine the magnitude of its acceleration at the instant  $t = 5$  s.

**12-131.** A boat is traveling along a circular path having a radius of 20 m. Determine the magnitude of the boat's acceleration when the speed is  $v = 5$  m/s and the rate of increase in the speed is  $\dot{v} = 2$  m/s $^2$ .

**\*12-132.** Starting from rest, a bicyclist travels around a horizontal circular path,  $\rho = 10$  m, at a speed of  $v = (0.09t^2 + 0.1t)$  m/s, where  $t$  is in seconds. Determine the magnitudes of his velocity and acceleration when he has traveled  $s = 3$  m.

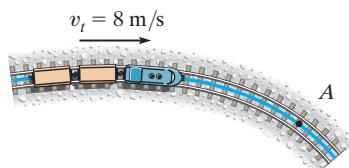
**12-133.** A particle travels around a circular path having a radius of 50 m. If it is initially traveling with a speed of 10 m/s and its speed then increases at a rate of  $\dot{v} = (0.05v)$  m/s $^2$ , determine the magnitude of the particle's acceleration four seconds later.

**12-134.** The motorcycle is traveling at a constant speed of 60 km/h. Determine the magnitude of its acceleration when it is at point A.



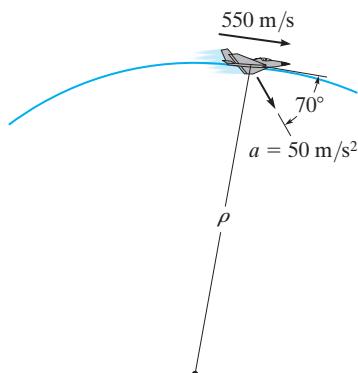
Prob. 12-134

**12-135.** When  $t = 0$ , the train has a speed of 8 m/s, which is increasing at  $0.5$  m/s $^2$ . Determine the magnitude of the acceleration of the engine when it reaches point A, at  $t = 20$  s. Here the radius of curvature of the tracks is  $\rho_A = 400$  m.



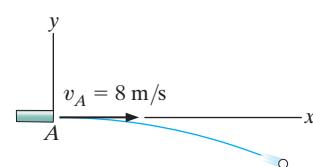
Prob. 12-135

**\*12-136.** At a given instant the jet plane has a speed of 550 m/s and an acceleration of 50 m/s $^2$  acting in the direction shown. Determine the rate of increase in the plane's speed, and also the radius of curvature  $\rho$  of the path.



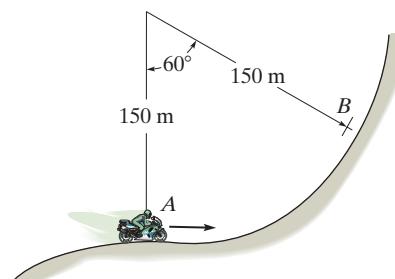
Prob. 12-136

**12-137.** The ball is ejected horizontally from the tube with a speed of 8 m/s. Find the equation of the path,  $y = f(x)$ , and then find the ball's velocity and the normal and tangential components of acceleration when  $t = 0.25$  s.



Prob. 12-137

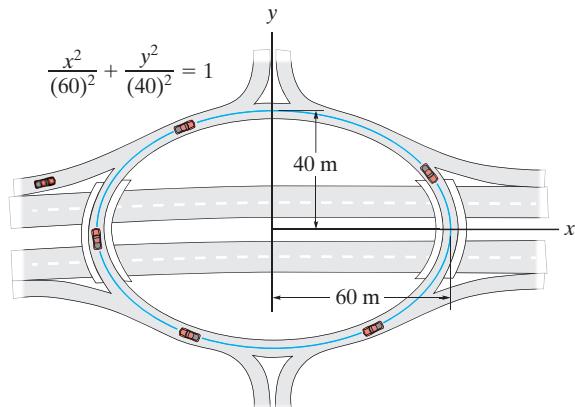
**12-138.** The motorcycle is traveling at 40 m/s when it is at A. If the speed is then decreased at  $\dot{v} = -(0.05s)$  m/s $^2$ , where  $s$  is in meters measured from A, determine its speed and acceleration when it reaches B.



Prob. 12-138

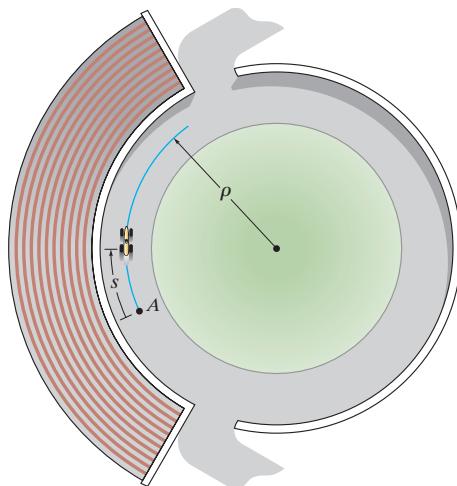
**12-139.** Cars move around the “traffic circle” which is in the shape of an ellipse. If the speed limit is posted at 60 km/h, determine the minimum acceleration experienced by the passengers.

\***12-140.** Cars move around the “traffic circle” which is in the shape of an ellipse. If the speed limit is posted at 60 km/h, determine the maximum acceleration experienced by the passengers.



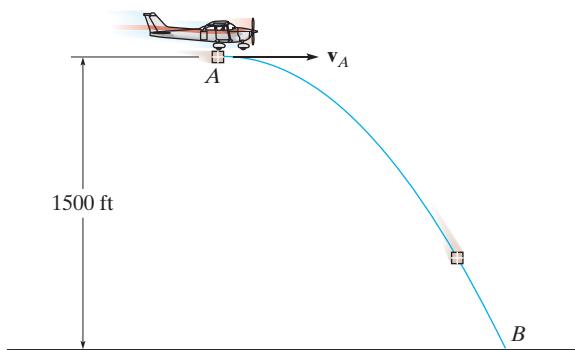
Probs. 12-139/140

**12-142.** The race car has an initial speed  $v_A = 15 \text{ m/s}$  at  $A$ . If it increases its speed along the circular track at the rate  $a_t = (0.4\text{s}) \text{ m/s}^2$ , where  $s$  is in meters, determine the time needed for the car to travel 20 m. Take  $\rho = 150 \text{ m}$ .



Prob. 12-142

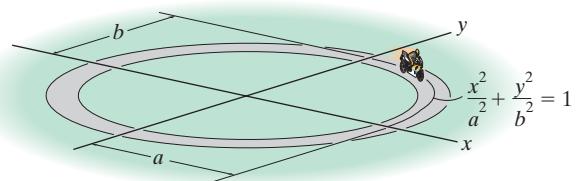
**12-141.** A package is dropped from the plane which is flying with a constant horizontal velocity of  $v_A = 150 \text{ ft/s}$ . Determine the normal and tangential components of acceleration and the radius of curvature of the path of motion (a) at the moment the package is released at  $A$ , where it has a horizontal velocity of  $v_A = 150 \text{ ft/s}$ , and (b) just before it strikes the ground at  $B$ .



Prob. 12-141

**12-143.** The motorcycle travels along the elliptical track at a constant speed  $v$ . Determine its greatest acceleration if  $a > b$ .

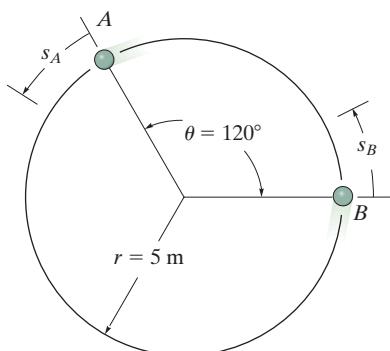
\***12-144.** The motorcycle travels along the elliptical track at a constant speed  $v$ . Determine its smallest acceleration if  $a > b$ .



Probs. 12-143/144

**12-145.** Particles *A* and *B* are traveling counter-clockwise around a circular track at a constant speed of 8 m/s. If at the instant shown the speed of *A* begins to increase by  $(a_t)_A = (0.4s_A)$  m/s<sup>2</sup>, where  $s_A$  is in meters, determine the distance measured counterclockwise along the track from *B* to *A* when  $t = 1$  s. What is the magnitude of the acceleration of each particle at this instant?

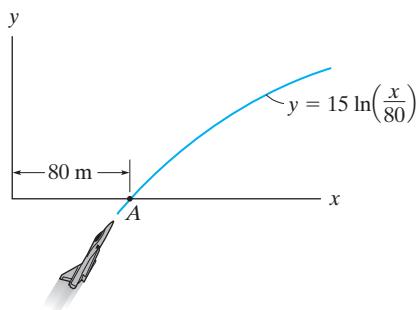
**12-146.** Particles *A* and *B* are traveling around a circular track at a speed of 8 m/s at the instant shown. If the speed of *B* is increasing by  $(a_t)_B = 4$  m/s<sup>2</sup>, and at the same instant *A* has an increase in speed of  $(a_t)_A = 0.8t$  m/s<sup>2</sup>, determine how long it takes for a collision to occur. What is the magnitude of the acceleration of each particle just before the collision occurs?



Probs. 12-145/146

**12-147.** The jet plane is traveling with a speed of 120 m/s which is decreasing at 40 m/s<sup>2</sup> when it reaches point *A*. Determine the magnitude of its acceleration when it is at this point. Also, specify the direction of flight, measured from the *x* axis.

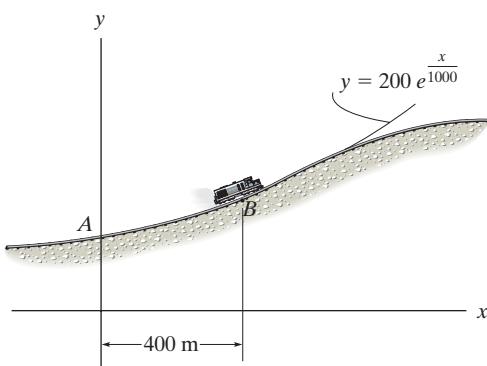
**\*12-148.** The jet plane is traveling with a constant speed of 110 m/s along the curved path. Determine the magnitude of the acceleration of the plane at the instant it reaches point *A* ( $y = 0$ ).



Probs. 12-147/148

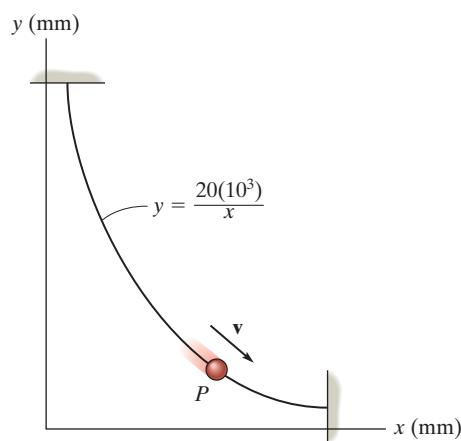
**12-149.** The train passes point *B* with a speed of 20 m/s which is decreasing at  $a_t = -0.5$  m/s<sup>2</sup>. Determine the magnitude of acceleration of the train at this point.

**12-150.** The train passes point *A* with a speed of 30 m/s and begins to decrease its speed at a constant rate of  $a_t = -0.25$  m/s<sup>2</sup>. Determine the magnitude of the acceleration of the train when it reaches point *B*, where  $s_{AB} = 412$  m.



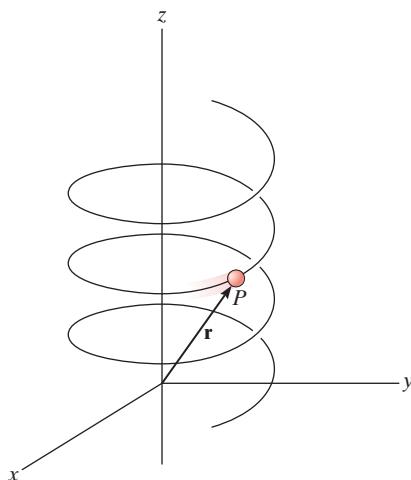
Probs. 12-149/150

**12-151.** The particle travels with a constant speed of 300 mm/s along the curve. Determine the particle's acceleration when it is located at point (200 mm, 100 mm) and sketch this vector on the curve.



Prob. 12-151

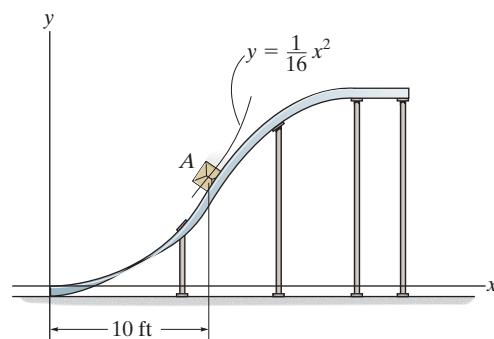
**\*12–152.** A particle  $P$  travels along an elliptical spiral path such that its position vector  $\mathbf{r}$  is defined by  $\mathbf{r} = \{2 \cos(0.1t)\mathbf{i} + 1.5 \sin(0.1t)\mathbf{j} + (2t)\mathbf{k}\}$  m, where  $t$  is in seconds and the arguments for the sine and cosine are given in radians. When  $t = 8$  s, determine the coordinate direction angles  $\alpha$ ,  $\beta$ , and  $\gamma$ , which the binormal axis to the osculating plane makes with the  $x$ ,  $y$ , and  $z$  axes. Hint: Solve for the velocity  $\mathbf{v}_P$  and acceleration  $\mathbf{a}_P$  of the particle in terms of their  $\mathbf{i}, \mathbf{j}, \mathbf{k}$  components. The binormal is parallel to  $\mathbf{v}_P \times \mathbf{a}_P$ . Why?



Prob. 12–152

**12–153.** The motion of a particle is defined by the equations  $x = (2t + t^2)$  m and  $y = (t^2)$  m, where  $t$  is in seconds. Determine the normal and tangential components of the particle's velocity and acceleration when  $t = 2$  s.

**12–154.** If the speed of the crate at  $A$  is 15 ft/s, which is increasing at a rate  $\dot{v} = 3$  ft/s<sup>2</sup>, determine the magnitude of the acceleration of the crate at this instant.



Prob. 12–154

## 12.8 Curvilinear Motion: Cylindrical Components

Sometimes the motion of the particle is constrained on a path that is best described using cylindrical coordinates. If motion is restricted to the plane, then polar coordinates are used.

**Polar Coordinates.** We can specify the location of the particle shown in Fig. 12–30a using a *radial coordinate*  $r$ , which extends outward from the fixed origin  $O$  to the particle, and a *transverse coordinate*  $\theta$ , which is the counterclockwise angle between a fixed reference line and the  $r$  axis. The angle is generally measured in degrees or radians, where  $1 \text{ rad} = 180^\circ/\pi$ . The positive directions of the  $r$  and  $\theta$  coordinates are defined by the unit vectors  $\mathbf{u}_r$  and  $\mathbf{u}_\theta$ , respectively. Here  $\mathbf{u}_r$  is in the direction of increasing  $r$  when  $\theta$  is held fixed, and  $\mathbf{u}_\theta$  is in a direction of increasing  $\theta$  when  $r$  is held fixed. Note that these directions are perpendicular to one another.

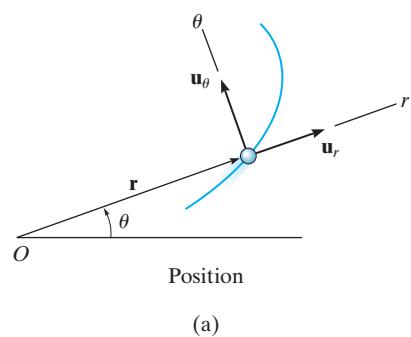
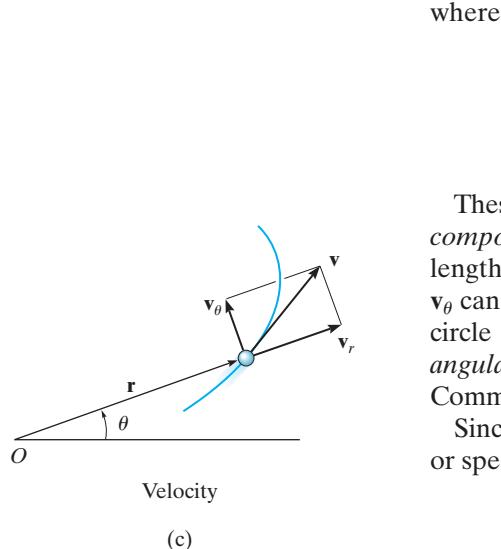
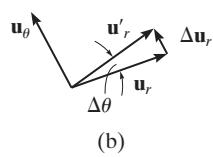
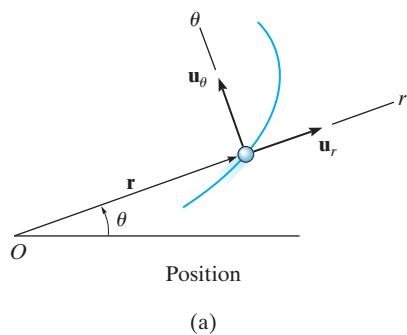


Fig. 12–30



**Position.** At any instant the position of the particle, Fig. 12-30a, is defined by the position vector

$$\mathbf{r} = r \mathbf{u}_r \quad (12-22)$$

**Velocity.** The instantaneous velocity  $\mathbf{v}$  is obtained by taking the time derivative of  $\mathbf{r}$ . Using a dot to represent the time derivative, we have

$$\mathbf{v} = \dot{\mathbf{r}} = \dot{r} \mathbf{u}_r + r \dot{\mathbf{u}}_r$$

To evaluate  $\dot{\mathbf{u}}_r$ , notice that  $\mathbf{u}_r$  only changes its direction with respect to time, since by definition the magnitude of this vector is always one unit. Hence, during the time  $\Delta t$ , a change  $\Delta r$  will not cause a change in the direction of  $\mathbf{u}_r$ ; however, a change  $\Delta\theta$  will cause  $\mathbf{u}_r$  to become  $\mathbf{u}'_r$ , where  $\mathbf{u}'_r = \mathbf{u}_r + \Delta\mathbf{u}_r$ , Fig. 12-30b. The time change in  $\mathbf{u}_r$  is then  $\Delta\mathbf{u}_r$ . For small angles  $\Delta\theta$  this vector has a magnitude  $\Delta u_r \approx 1(\Delta\theta)$  and acts in the  $\mathbf{u}_\theta$  direction. Therefore,  $\Delta\mathbf{u}_r = \Delta\theta\mathbf{u}_\theta$ , and so

$$\dot{\mathbf{u}}_r = \lim_{\Delta t \rightarrow 0} \frac{\Delta \mathbf{u}_r}{\Delta t} = \left( \lim_{\Delta t \rightarrow 0} \frac{\Delta\theta}{\Delta t} \right) \mathbf{u}_\theta$$

$$\dot{\mathbf{u}}_r = \dot{\theta} \mathbf{u}_\theta \quad (12-23)$$

Substituting into the above equation, the velocity can be written in component form as

$$\mathbf{v} = v_r \mathbf{u}_r + v_\theta \mathbf{u}_\theta \quad (12-24)$$

where

$$\boxed{\begin{aligned} v_r &= \dot{r} \\ v_\theta &= r\dot{\theta} \end{aligned}} \quad (12-25)$$

These components are shown graphically in Fig. 12-30c. The *radial component*  $v_r$  is a measure of the rate of increase or decrease in the length of the radial coordinate, i.e.,  $\dot{r}$ ; whereas the *transverse component*  $v_\theta$  can be interpreted as the rate of motion along the circumference of a circle having a radius  $r$ . In particular, the term  $\dot{\theta} = d\theta/dt$  is called the *angular velocity*, since it indicates the time rate of change of the angle  $\theta$ . Common units used for this measurement are rad/s.

Since  $\mathbf{v}_r$  and  $\mathbf{v}_\theta$  are mutually perpendicular, the *magnitude* of velocity or speed is simply the positive value of

$$v = \sqrt{(\dot{r})^2 + (r\dot{\theta})^2} \quad (12-26)$$

and the *direction* of  $\mathbf{v}$  is, of course, tangent to the path, Fig. 12-30c.

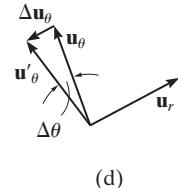
Fig. 12-30 (cont.)

**Acceleration.** Taking the time derivatives of Eq. 12–24, using Eqs. 12–25, we obtain the particle's instantaneous acceleration,

$$\mathbf{a} = \dot{\mathbf{v}} = \dot{r}\mathbf{u}_r + r\dot{\theta}\mathbf{u}_\theta + r\ddot{\theta}\mathbf{u}_\theta + r\dot{\theta}^2\mathbf{u}_\theta$$

To evaluate  $\dot{\mathbf{u}}_\theta$ , it is necessary only to find the change in the direction of  $\mathbf{u}_\theta$  since its magnitude is always unity. During the time  $\Delta t$ , a change  $\Delta r$  will not change the direction of  $\mathbf{u}_\theta$ , however, a change  $\Delta\theta$  will cause  $\mathbf{u}_\theta$  to become  $\mathbf{u}'_\theta$ , where  $\mathbf{u}'_\theta = \mathbf{u}_\theta + \Delta\mathbf{u}_\theta$ , Fig. 12–30d. The time change in  $\mathbf{u}_\theta$  is thus  $\Delta\mathbf{u}_\theta$ . For small angles this vector has a magnitude  $\Delta u_\theta \approx 1(\Delta\theta)$  and acts in the  $-\mathbf{u}_r$  direction; i.e.,  $\Delta\mathbf{u}_\theta = -\Delta\theta\mathbf{u}_r$ . Thus,

$$\begin{aligned}\dot{\mathbf{u}}_\theta &= \lim_{\Delta t \rightarrow 0} \frac{\Delta \mathbf{u}_\theta}{\Delta t} = -\left(\lim_{\Delta t \rightarrow 0} \frac{\Delta\theta}{\Delta t}\right)\mathbf{u}_r \\ \dot{\mathbf{u}}_\theta &= -\dot{\theta}\mathbf{u}_r\end{aligned}\quad (12-27)$$



Substituting this result and Eq. 12–23 into the above equation for  $\mathbf{a}$ , we can write the acceleration in component form as

$$\mathbf{a} = a_r\mathbf{u}_r + a_\theta\mathbf{u}_\theta \quad (12-28)$$

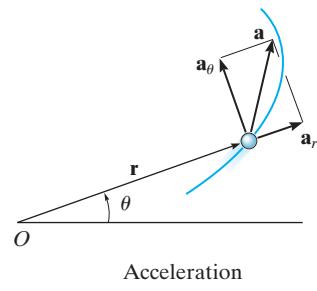
where

$$\begin{aligned}a_r &= \ddot{r} - r\dot{\theta}^2 \\ a_\theta &= r\ddot{\theta} + 2r\dot{\theta}\end{aligned}\quad (12-29)$$

The term  $\ddot{\theta} = d^2\theta/dt^2 = d/dt(d\theta/dt)$  is called the *angular acceleration* since it measures the change made in the angular velocity during an instant of time. Units for this measurement are  $\text{rad/s}^2$ .

Since  $\mathbf{a}_r$  and  $\mathbf{a}_\theta$  are always perpendicular, the *magnitude* of acceleration is simply the positive value of

$$a = \sqrt{(\ddot{r} - r\dot{\theta}^2)^2 + (r\ddot{\theta} + 2r\dot{\theta})^2} \quad (12-30)$$



The *direction* is determined from the vector addition of its two components. In general,  $\mathbf{a}$  will *not* be tangent to the path, Fig. 12–30e.

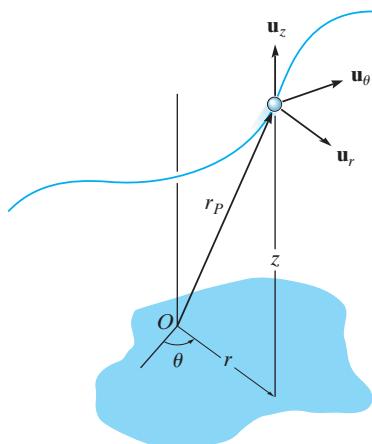


Fig. 12-31



The spiral motion of this girl can be followed by using cylindrical components. Here the radial coordinate  $r$  is constant, the transverse coordinate  $\theta$  will increase with time as the girl rotates about the vertical, and her altitude  $z$  will decrease with time. (© R.C. Hibbeler)

**Cylindrical Coordinates.** If the particle moves along a space curve as shown in Fig. 12-31, then its location may be specified by the three *cylindrical coordinates*,  $r$ ,  $\theta$ ,  $z$ . The  $z$  coordinate is identical to that used for rectangular coordinates. Since the unit vector defining its direction,  $\mathbf{u}_z$ , is constant, the time derivatives of this vector are zero, and therefore the position, velocity, and acceleration of the particle can be written in terms of its cylindrical coordinates as follows:

$$\mathbf{r}_P = r\mathbf{u}_r + z\mathbf{u}_z \quad (12-31)$$

$$\mathbf{v} = \dot{r}\mathbf{u}_r + r\dot{\theta}\mathbf{u}_\theta + \dot{z}\mathbf{u}_z \quad (12-32)$$

**Time Derivatives.** The above equations require that we obtain the time derivatives  $\dot{r}$ ,  $\ddot{r}$ ,  $\dot{\theta}$ , and  $\ddot{\theta}$  in order to evaluate the  $r$  and  $\theta$  components of  $\mathbf{v}$  and  $\mathbf{a}$ . Two types of problems generally occur:

1. If the polar coordinates are specified as time parametric equations,  $r = r(t)$  and  $\theta = \theta(t)$ , then the time derivatives can be found directly.
2. If the time-parametric equations are not given, then the path  $r = f(\theta)$  must be known. Using the chain rule of calculus we can then find the relation between  $\dot{r}$  and  $\dot{\theta}$ , and between  $\ddot{r}$  and  $\ddot{\theta}$ . Application of the chain rule, along with some examples, is explained in Appendix C.

### Procedure for Analysis

#### Coordinate System.

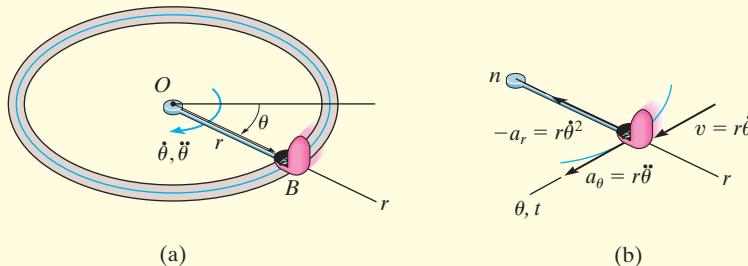
- Polar coordinates are a suitable choice for solving problems when data regarding the angular motion of the radial coordinate  $r$  is given to describe the particle's motion. Also, some paths of motion can conveniently be described in terms of these coordinates.
- To use polar coordinates, the origin is established at a fixed point, and the radial line  $r$  is directed to the particle.
- The transverse coordinate  $\theta$  is measured from a fixed reference line to the radial line.

#### Velocity and Acceleration.

- Once  $r$  and the four time derivatives  $\dot{r}$ ,  $\ddot{r}$ ,  $\dot{\theta}$ , and  $\ddot{\theta}$  have been evaluated at the instant considered, their values can be substituted into Eqs. 12-25 and 12-29 to obtain the radial and transverse components of  $\mathbf{v}$  and  $\mathbf{a}$ .
- If it is necessary to take the time derivatives of  $r = f(\theta)$ , then the chain rule of calculus must be used. See Appendix C.
- Motion in three dimensions requires a simple extension of the above procedure to include  $\dot{z}$  and  $\ddot{z}$ .

**EXAMPLE | 12.17**

The amusement park ride shown in Fig. 12–32a consists of a chair that is rotating in a horizontal circular path of radius  $r$  such that the arm  $OB$  has an angular velocity  $\dot{\theta}$  and angular acceleration  $\ddot{\theta}$ . Determine the radial and transverse components of velocity and acceleration of the passenger. Neglect his size in the calculation.

**Fig. 12–32****SOLUTION**

**Coordinate System.** Since the angular motion of the arm is reported, polar coordinates are chosen for the solution, Fig. 12–32a. Here  $\theta$  is not related to  $r$ , since the radius is constant for all  $\theta$ .

**Velocity and Acceleration.** It is first necessary to specify the first and second time derivatives of  $r$  and  $\theta$ . Since  $r$  is *constant*, we have

$$r = r \quad \dot{r} = 0 \quad \ddot{r} = 0$$

Thus,

$$v_r = \dot{r} = 0 \quad \text{Ans.}$$

$$v_\theta = r\dot{\theta} \quad \text{Ans.}$$

$$a_r = \ddot{r} - r\dot{\theta}^2 = -r\dot{\theta}^2 \quad \text{Ans.}$$

$$a_\theta = r\ddot{\theta} + 2\dot{r}\dot{\theta} = r\ddot{\theta} \quad \text{Ans.}$$

These results are shown in Fig. 12–32b.

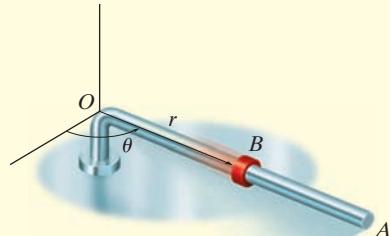
**NOTE:** The  $n, t$  axes are also shown in Fig. 12–32b, which in this special case of circular motion happen to be *collinear* with the  $r$  and  $\theta$  axes, respectively. Since  $v = v_\theta = v_t = r\dot{\theta}$ , then by comparison,

$$-a_r = a_n = \frac{v^2}{r} = \frac{(r\dot{\theta})^2}{r} = r\dot{\theta}^2$$

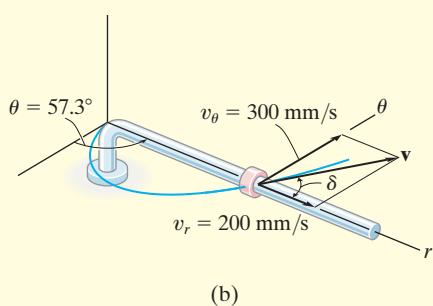
$$a_\theta = a_t = \frac{dv}{dt} = \frac{d}{dt}(r\dot{\theta}) = \frac{dr}{dt}\dot{\theta} + r\frac{d\dot{\theta}}{dt} = 0 + r\ddot{\theta}$$

## EXAMPLE 12.18

The rod  $OA$  in Fig. 12-33a rotates in the horizontal plane such that  $\theta = (t^3)$  rad. At the same time, the collar  $B$  is sliding outward along  $OA$  so that  $r = (100t^2)$  mm. If in both cases  $t$  is in seconds, determine the velocity and acceleration of the collar when  $t = 1$  s.



(a)



(b)

**SOLUTION**

**Coordinate System.** Since time-parametric equations of the path are given, it is not necessary to relate  $r$  to  $\theta$ .

**Velocity and Acceleration.** Determining the time derivatives and evaluating them when  $t = 1$  s, we have

$$r = 100t^2 \Big|_{t=1\text{ s}} = 100\text{ mm} \quad \theta = t^3 \Big|_{t=1\text{ s}} = 1 \text{ rad} = 57.3^\circ$$

$$\dot{r} = 200t \Big|_{t=1\text{ s}} = 200 \text{ mm/s} \quad \dot{\theta} = 3t^2 \Big|_{t=1\text{ s}} = 3 \text{ rad/s}$$

$$\ddot{r} = 200 \Big|_{t=1\text{ s}} = 200 \text{ mm/s}^2 \quad \ddot{\theta} = 6t \Big|_{t=1\text{ s}} = 6 \text{ rad/s}^2.$$

As shown in Fig. 12-33b,

$$\begin{aligned} \mathbf{v} &= \dot{r}\mathbf{u}_r + r\dot{\theta}\mathbf{u}_\theta \\ &= 200\mathbf{u}_r + 100(3)\mathbf{u}_\theta = \{200\mathbf{u}_r + 300\mathbf{u}_\theta\} \text{ mm/s} \end{aligned}$$

The magnitude of  $\mathbf{v}$  is

$$v = \sqrt{(200)^2 + (300)^2} = 361 \text{ mm/s} \quad \text{Ans.}$$

$$\delta = \tan^{-1}\left(\frac{300}{200}\right) = 56.3^\circ \quad \delta + 57.3^\circ = 114^\circ \quad \text{Ans.}$$

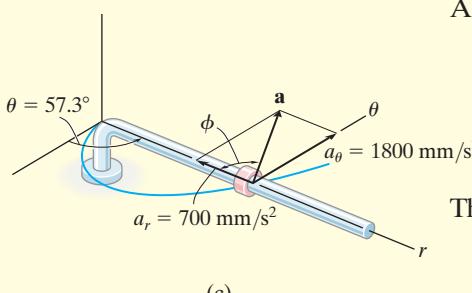
As shown in Fig. 12-33c,

$$\begin{aligned} \mathbf{a} &= (\ddot{r} - r\dot{\theta}^2)\mathbf{u}_r + (r\ddot{\theta} + 2r\dot{\theta})\mathbf{u}_\theta \\ &= [200 - 100(3)^2]\mathbf{u}_r + [100(6) + 2(200)3]\mathbf{u}_\theta \\ &= \{-700\mathbf{u}_r + 1800\mathbf{u}_\theta\} \text{ mm/s}^2 \end{aligned}$$

The magnitude of  $\mathbf{a}$  is

$$a = \sqrt{(-700)^2 + (1800)^2} = 1930 \text{ mm/s}^2 \quad \text{Ans.}$$

$$\phi = \tan^{-1}\left(\frac{1800}{700}\right) = 68.7^\circ \quad (180^\circ - \phi) + 57.3^\circ = 169^\circ \quad \text{Ans.}$$



(c)

Fig. 12-33

**NOTE:** The velocity is tangent to the path; however, the acceleration is directed within the curvature of the path, as expected.

**EXAMPLE | 12.19**

The searchlight in Fig. 12-34a casts a spot of light along the face of a wall that is located 100 m from the searchlight. Determine the magnitudes of the velocity and acceleration at which the spot appears to travel across the wall at the instant  $\theta = 45^\circ$ . The searchlight rotates at a constant rate of  $\dot{\theta} = 4 \text{ rad/s}$ .

**SOLUTION**

**Coordinate System.** Polar coordinates will be used to solve this problem since the angular rate of the searchlight is given. To find the necessary time derivatives it is first necessary to relate  $r$  to  $\theta$ . From Fig. 12-34a,

$$r = 100/\cos \theta = 100 \sec \theta$$

**Velocity and Acceleration.** Using the chain rule of calculus, noting that  $d(\sec \theta) = \sec \theta \tan \theta d\theta$ , and  $d(\tan \theta) = \sec^2 \theta d\theta$ , we have

$$\begin{aligned}\dot{r} &= 100(\sec \theta \tan \theta)\dot{\theta} \\ \ddot{r} &= 100(\sec \theta \tan \theta)\dot{\theta}(\tan \theta)\dot{\theta} + 100 \sec \theta (\sec^2 \theta)\dot{\theta}(\dot{\theta}) \\ &\quad + 100 \sec \theta \tan \theta(\ddot{\theta}) \\ &= 100 \sec \theta \tan^2 \theta(\dot{\theta})^2 + 100 \sec^3 \theta(\dot{\theta})^2 + 100(\sec \theta \tan \theta)\ddot{\theta}\end{aligned}$$

Since  $\dot{\theta} = 4 \text{ rad/s} = \text{constant}$ , then  $\ddot{\theta} = 0$ , and the above equations, when  $\theta = 45^\circ$ , become

$$\begin{aligned}r &= 100 \sec 45^\circ = 141.4 \\ \dot{r} &= 400 \sec 45^\circ \tan 45^\circ = 565.7 \\ \ddot{r} &= 1600 (\sec 45^\circ \tan^2 45^\circ + \sec^3 45^\circ) = 6788.2\end{aligned}$$

As shown in Fig. 12-34b,

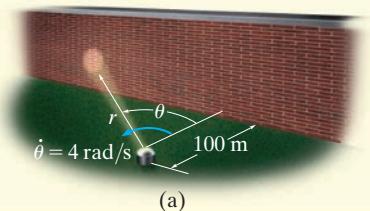
$$\begin{aligned}\mathbf{v} &= \dot{r}\mathbf{u}_r + r\dot{\theta}\mathbf{u}_\theta \\ &= 565.7\mathbf{u}_r + 141.4(4)\mathbf{u}_\theta \\ &= \{565.7\mathbf{u}_r + 565.7\mathbf{u}_\theta\} \text{ m/s} \\ v &= \sqrt{v_r^2 + v_\theta^2} = \sqrt{(565.7)^2 + (565.7)^2} \\ &= 800 \text{ m/s}\end{aligned}$$

*Ans.*

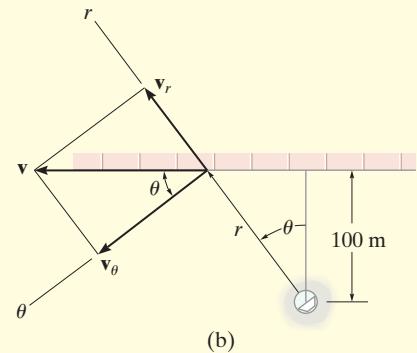
As shown in Fig. 12-34c,

$$\begin{aligned}\mathbf{a} &= (\ddot{r} - r\dot{\theta}^2)\mathbf{u}_r + (r\ddot{\theta} + 2\dot{r}\dot{\theta})\mathbf{u}_\theta \\ &= [6788.2 - 141.4(4)^2]\mathbf{u}_r + [141.4(0) + 2(565.7)4]\mathbf{u}_\theta \\ &= \{4525.5\mathbf{u}_r + 4525.5\mathbf{u}_\theta\} \text{ m/s}^2 \\ a &= \sqrt{a_r^2 + a_\theta^2} = \sqrt{(4525.5)^2 + (4525.5)^2} \\ &= 6400 \text{ m/s}^2\end{aligned}$$

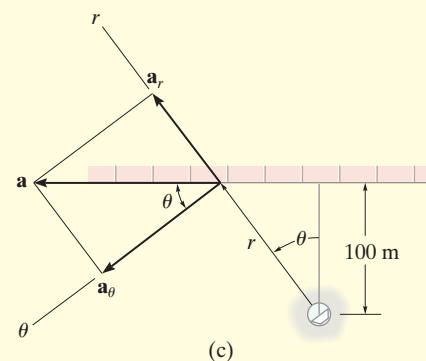
*Ans.*



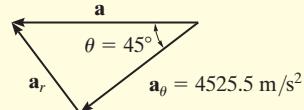
(a)



(b)



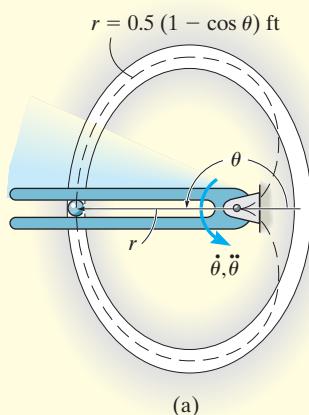
(c)



(d)

**Fig. 12-34**

## EXAMPLE 12.20



Due to the rotation of the forked rod, the ball in Fig. 12-35a travels around the slotted path, a portion of which is in the shape of a cardioid,  $r = 0.5(1 - \cos \theta)$  ft, where  $\theta$  is in radians. If the ball's velocity is  $v = 4$  ft/s and its acceleration is  $a = 30$  ft/s<sup>2</sup> at the instant  $\theta = 180^\circ$ , determine the angular velocity  $\dot{\theta}$  and angular acceleration  $\ddot{\theta}$  of the fork.

## SOLUTION

**Coordinate System.** This path is most unusual, and mathematically it is best expressed using polar coordinates, as done here, rather than rectangular coordinates. Also, since  $\dot{\theta}$  and  $\ddot{\theta}$  must be determined, then  $r, \theta$  coordinates are an obvious choice.

**Velocity and Acceleration.** The time derivatives of  $r$  and  $\theta$  can be determined using the chain rule.

$$r = 0.5(1 - \cos \theta)$$

$$\dot{r} = 0.5(\sin \theta)\dot{\theta}$$

$$\ddot{r} = 0.5(\cos \theta)\dot{\theta}\dot{\theta} + 0.5(\sin \theta)\ddot{\theta}$$

Evaluating these results at  $\theta = 180^\circ$ , we have

$$r = 1 \text{ ft} \quad \dot{r} = 0 \quad \ddot{r} = -0.5\dot{\theta}^2$$

Since  $v = 4$  ft/s, using Eq. 12-26 to determine  $\dot{\theta}$  yields

$$v = \sqrt{(\dot{r})^2 + (r\dot{\theta})^2}$$

$$4 = \sqrt{(0)^2 + (1\dot{\theta})^2}$$

$$\dot{\theta} = 4 \text{ rad/s}$$

*Ans.*

In a similar manner,  $\ddot{\theta}$  can be found using Eq. 12-30.

$$a = \sqrt{(\ddot{r} - r\dot{\theta}^2)^2 + (r\ddot{\theta} + 2r\dot{\theta})^2}$$

$$30 = \sqrt{[-0.5(4)^2 - 1(4)^2]^2 + [1\ddot{\theta} + 2(0)(4)]^2}$$

$$(30)^2 = (-24)^2 + \ddot{\theta}^2$$

$$\ddot{\theta} = 18 \text{ rad/s}^2$$

*Ans.*

Vectors **a** and **v** are shown in Fig. 12-35b.

**NOTE:** At this location, the  $\theta$  and  $t$  (tangential) axes will coincide. The  $+n$  (normal) axis is directed to the right, opposite to  $+r$ .

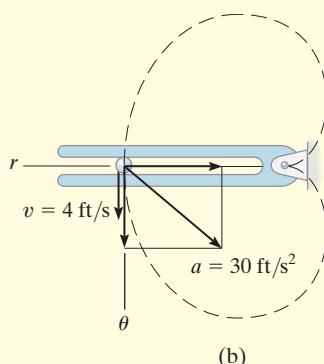
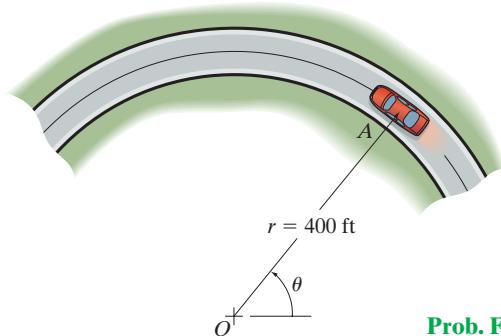


Fig. 12-35

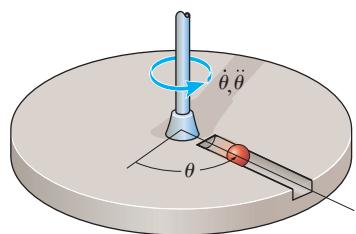
## FUNDAMENTAL PROBLEMS

**F12–33.** The car has a speed of 55 ft/s. Determine the angular velocity  $\dot{\theta}$  of the radial line  $OA$  at this instant.



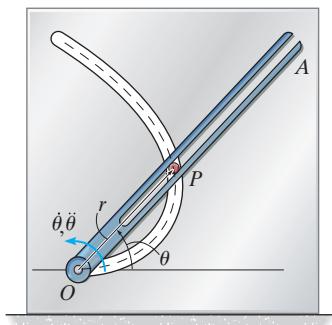
Prob. F12–33

**F12–34.** The platform is rotating about the vertical axis such that at any instant its angular position is  $\theta = (4t^{3/2})$  rad, where  $t$  is in seconds. A ball rolls outward along the radial groove so that its position is  $r = (0.1t^3)$  m, where  $t$  is in seconds. Determine the magnitudes of the velocity and acceleration of the ball when  $t = 1.5$  s.



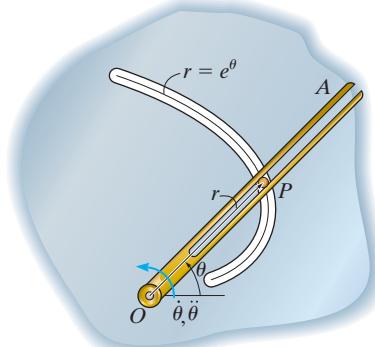
Prob. F12–34

**F12–35.** Peg  $P$  is driven by the fork link  $OA$  along the curved path described by  $r = (2\theta)$  ft. At the instant  $\theta = \pi/4$  rad, the angular velocity and angular acceleration of the link are  $\dot{\theta} = 3$  rad/s and  $\ddot{\theta} = 1$  rad/s<sup>2</sup>. Determine the magnitude of the peg's acceleration at this instant.



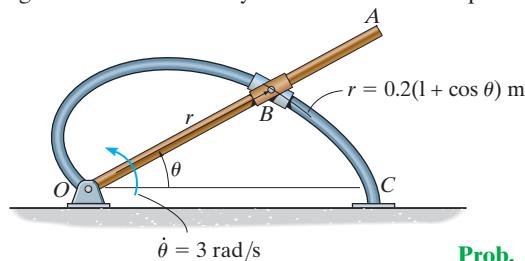
Prob. F12–35

**F12–36.** Peg  $P$  is driven by the forked link  $OA$  along the path described by  $r = e^\theta$ , where  $r$  is in meters. When  $\theta = \frac{\pi}{4}$  rad, the link has an angular velocity and angular acceleration of  $\dot{\theta} = 2$  rad/s and  $\ddot{\theta} = 4$  rad/s<sup>2</sup>. Determine the radial and transverse components of the peg's acceleration at this instant.



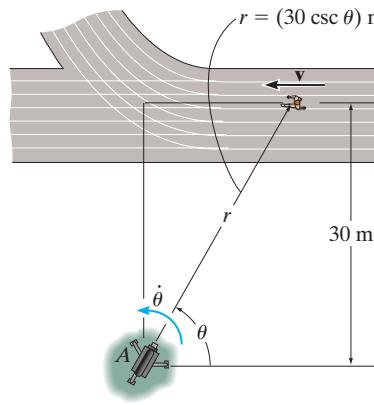
Prob. F12–36

**F12–37.** The collars are pin connected at  $B$  and are free to move along rod  $OA$  and the curved guide  $OC$  having the shape of a cardioid,  $r = [0.2(1 + \cos \theta)]$  m. At  $\theta = 30^\circ$ , the angular velocity of  $OA$  is  $\dot{\theta} = 3$  rad/s. Determine the magnitude of the velocity of the collars at this point.



Prob. F12–37

**F12–38.** At the instant  $\theta = 45^\circ$ , the athlete is running with a constant speed of 2 m/s. Determine the angular velocity at which the camera must turn in order to follow the motion.



Prob. F12–38

## PROBLEMS

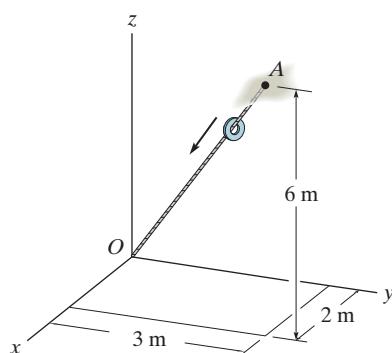
**12-155.** A particle is moving along a circular path having a radius of 4 in. such that its position as a function of time is given by  $\theta = \cos 2t$ , where  $\theta$  is in radians and  $t$  is in seconds. Determine the magnitude of the acceleration of the particle when  $\theta = 30^\circ$ .

**\*12-156.** For a short time a rocket travels up and to the right at a constant speed of 800 m/s along the parabolic path  $y = 600 - 35x^2$ . Determine the radial and transverse components of velocity of the rocket at the instant  $\theta = 60^\circ$ , where  $\theta$  is measured counterclockwise from the  $x$  axis.

**12-157.** A particle moves along a path defined by polar coordinates  $r = (2e^t)$  ft and  $\theta = (8t^2)$  rad, where  $t$  is in seconds. Determine the components of its velocity and acceleration when  $t = 1$  s.

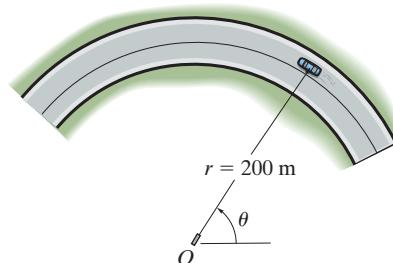
**12-158.** An airplane is flying in a straight line with a velocity of 200 mi/h and an acceleration of  $3 \text{ mi/h}^2$ . If the propeller has a diameter of 6 ft and is rotating at a constant angular rate of 120 rad/s, determine the magnitudes of velocity and acceleration of a particle located on the tip of the propeller.

**12-159.** The small washer is sliding down the cord  $OA$ . When it is at the midpoint, its speed is 28 m/s and its acceleration is  $7 \text{ m/s}^2$ . Express the velocity and acceleration of the washer at this point in terms of its cylindrical components.



Prob. 12-159

**\*12-160.** A radar gun at  $O$  rotates with the angular velocity of  $\dot{\theta} = 0.1 \text{ rad/s}$  and angular acceleration of  $\ddot{\theta} = 0.025 \text{ rad/s}^2$ , at the instant  $\theta = 45^\circ$ , as it follows the motion of the car traveling along the circular road having a radius of  $r = 200 \text{ m}$ . Determine the magnitudes of velocity and acceleration of the car at this instant.



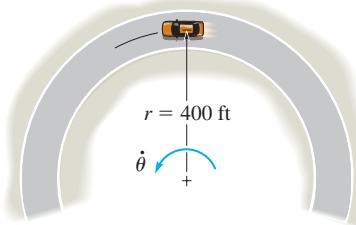
Prob. 12-160

**12-161.** If a particle moves along a path such that  $r = (2 \cos t)$  ft and  $\theta = (t/2)$  rad, where  $t$  is in seconds, plot the path  $r = f(\theta)$  and determine the particle's radial and transverse components of velocity and acceleration.

**12-162.** If a particle moves along a path such that  $r = (e^{at})$  m and  $\theta = t$ , where  $t$  is in seconds, plot the path  $r = f(\theta)$ , and determine the particle's radial and transverse components of velocity and acceleration.

**12-163.** The car travels along the circular curve having a radius  $r = 400 \text{ ft}$ . At the instant shown, its angular rate of rotation is  $\dot{\theta} = 0.025 \text{ rad/s}$ , which is decreasing at the rate  $\ddot{\theta} = -0.008 \text{ rad/s}^2$ . Determine the radial and transverse components of the car's velocity and acceleration at this instant and sketch these components on the curve.

**\*12-164.** The car travels along the circular curve of radius  $r = 400 \text{ ft}$  with a constant speed of  $v = 30 \text{ ft/s}$ . Determine the angular rate of rotation  $\dot{\theta}$  of the radial line  $r$  and the magnitude of the car's acceleration.

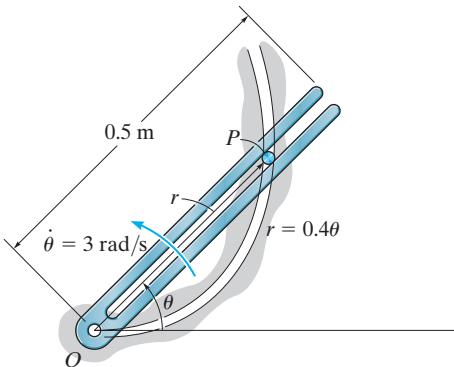


Probs. 12-163/164

**12–165.** The time rate of change of acceleration is referred to as the *jerk*, which is often used as a means of measuring passenger discomfort. Calculate this vector,  $\ddot{\mathbf{a}}$ , in terms of its cylindrical components, using Eq. 12–32.

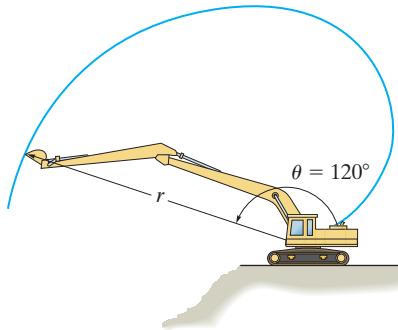
**12–166.** A particle is moving along a circular path having a radius of 6 in. such that its position as a function of time is given by  $\theta = \sin 3t$ , where  $\theta$  and the argument for the sine are in radians, and  $t$  is in seconds. Determine the magnitude of the acceleration of the particle at  $\theta = 30^\circ$ . The particle starts from rest at  $\theta = 0^\circ$ .

**12–167.** The slotted link is pinned at  $O$ , and as a result of the constant angular velocity  $\dot{\theta} = 3 \text{ rad/s}$  it drives the peg  $P$  for a short distance along the spiral guide  $r = (0.4\theta) \text{ m}$ , where  $\theta$  is in radians. Determine the radial and transverse components of the velocity and acceleration of  $P$  at the instant  $\theta = \pi/3 \text{ rad}$ .



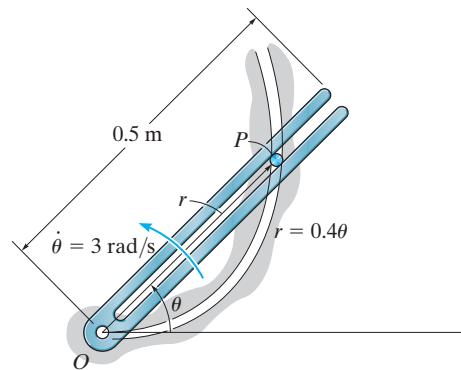
Prob. 12–167

\***12–168.** For a short time the bucket of the backhoe traces the path of the cardioid  $r = 25(1 - \cos \theta) \text{ ft}$ . Determine the magnitudes of the velocity and acceleration of the bucket when  $\theta = 120^\circ$  if the boom is rotating with an angular velocity of  $\dot{\theta} = 2 \text{ rad/s}$  and an angular acceleration of  $\ddot{\theta} = 0.2 \text{ rad/s}^2$  at the instant shown.



Prob. 12–168

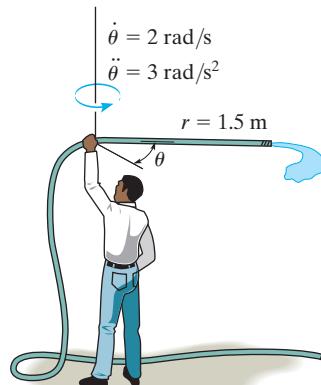
**12–169.** The slotted link is pinned at  $O$ , and as a result of the constant angular velocity  $\dot{\theta} = 3 \text{ rad/s}$  it drives the peg  $P$  for a short distance along the spiral guide  $r = (0.4\theta) \text{ m}$ , where  $\theta$  is in radians. Determine the velocity and acceleration of the particle at the instant it leaves the slot in the link, i.e., when  $r = 0.5 \text{ m}$ .



Prob. 12–169

**12–170.** A particle moves in the  $x-y$  plane such that its position is defined by  $r = \{2\mathbf{i} + 4t^2\mathbf{j}\} \text{ ft}$ , where  $t$  is in seconds. Determine the radial and transverse components of the particle's velocity and acceleration when  $t = 2 \text{ s}$ .

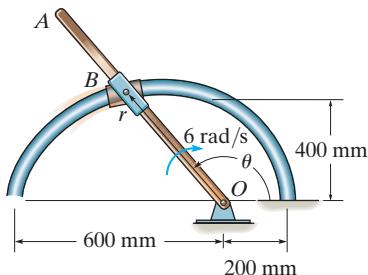
**12–171.** At the instant shown, the man is twirling a hose over his head with an angular velocity  $\dot{\theta} = 2 \text{ rad/s}$  and an angular acceleration  $\ddot{\theta} = 3 \text{ rad/s}^2$ . If it is assumed that the hose lies in a horizontal plane, and water is flowing through it at a constant rate of 3 m/s, determine the magnitudes of the velocity and acceleration of a water particle as it exits the open end,  $r = 1.5 \text{ m}$ .



Prob. 12–171

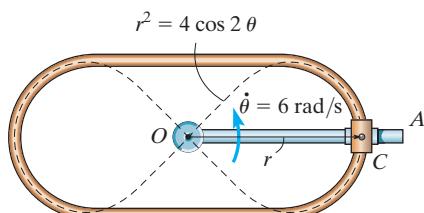
**\*12-172.** The rod  $OA$  rotates clockwise with a constant angular velocity of  $6 \text{ rad/s}$ . Two pin-connected slider blocks, located at  $B$ , move freely on  $OA$  and the curved rod whose shape is a limaçon described by the equation  $r = 200(2 - \cos \theta) \text{ mm}$ . Determine the speed of the slider blocks at the instant  $\theta = 150^\circ$ .

**12-173.** Determine the magnitude of the acceleration of the slider blocks in Prob. 12-172 when  $\theta = 150^\circ$ .



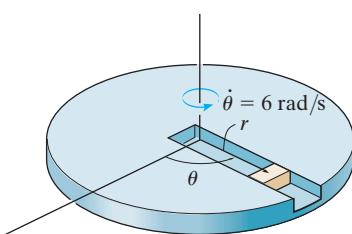
Probs. 12-172/173

**12-174.** A double collar  $C$  is pin connected together such that one collar slides over a fixed rod and the other slides over a rotating rod. If the geometry of the fixed rod for a short distance can be defined by a lemniscate,  $r^2 = (4 \cos 2\theta) \text{ ft}^2$ , determine the collar's radial and transverse components of velocity and acceleration at the instant  $\theta = 0^\circ$  as shown. Rod  $OA$  is rotating at a constant rate of  $\dot{\theta} = 6 \text{ rad/s}$ .



Prob. 12-174

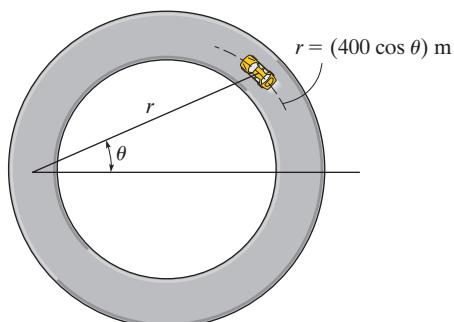
**12-175.** A block moves outward along the slot in the platform with a speed of  $\dot{r} = (4t) \text{ m/s}$ , where  $t$  is in seconds. The platform rotates at a constant rate of  $6 \text{ rad/s}$ . If the block starts from rest at the center, determine the magnitudes of its velocity and acceleration when  $t = 1 \text{ s}$ .



Prob. 12-175

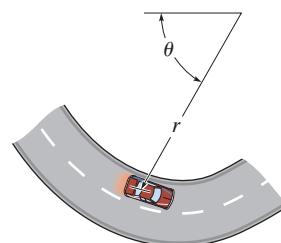
**\*12-176.** The car travels around the circular track with a constant speed of  $20 \text{ m/s}$ . Determine the car's radial and transverse components of velocity and acceleration at the instant  $\theta = \pi/4 \text{ rad}$ .

**12-177.** The car travels around the circular track such that its transverse component is  $\theta = (0.006t^2) \text{ rad}$ , where  $t$  is in seconds. Determine the car's radial and transverse components of velocity and acceleration at the instant  $t = 4 \text{ s}$ .



Probs. 12-176/177

**12-178.** The car travels along a road which for a short distance is defined by  $r = (200/\theta) \text{ ft}$ , where  $\theta$  is in radians. If it maintains a constant speed of  $v = 35 \text{ ft/s}$ , determine the radial and transverse components of its velocity when  $\theta = \pi/3 \text{ rad}$ .



Prob. 12-178

**12-179.** A horse on the merry-go-round moves according to the equations  $r = 8 \text{ ft}$ ,  $\theta = (0.6t) \text{ rad}$ , and  $z = (1.5 \sin \theta) \text{ ft}$ , where  $t$  is in seconds. Determine the cylindrical components of the velocity and acceleration of the horse when  $t = 4 \text{ s}$ .

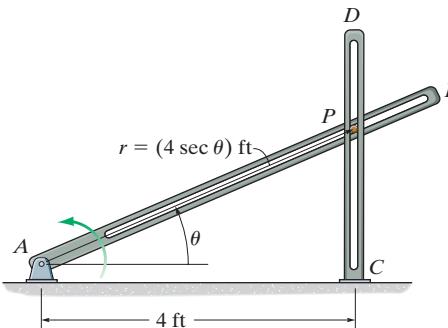
**\*12-180.** A horse on the merry-go-round moves according to the equations  $r = 8 \text{ ft}$ ,  $\dot{\theta} = 2 \text{ rad/s}$  and  $z = (1.5 \sin \theta) \text{ ft}$ , where  $t$  is in seconds. Determine the maximum and minimum magnitudes of the velocity and acceleration of the horse during the motion.



Probs. 12-179/180

**12-181.** If the slotted arm  $AB$  rotates counterclockwise with a constant angular velocity of  $\dot{\theta} = 2 \text{ rad/s}$ , determine the magnitudes of the velocity and acceleration of peg  $P$  at  $\theta = 30^\circ$ . The peg is constrained to move in the slots of the fixed bar  $CD$  and rotating bar  $AB$ .

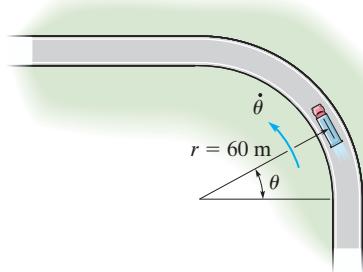
**12-182.** The peg is constrained to move in the slots of the fixed bar  $CD$  and rotating bar  $AB$ . When  $\theta = 30^\circ$ , the angular velocity and angular acceleration of arm  $AB$  are  $\dot{\theta} = 2 \text{ rad/s}$  and  $\ddot{\theta} = 3 \text{ rad/s}^2$ , respectively. Determine the magnitudes of the velocity and acceleration of the peg  $P$  at this instant.



Probs. 12-181/182

**12-183.** A truck is traveling along the horizontal circular curve of radius  $r = 60 \text{ m}$  with a constant speed  $v = 20 \text{ m/s}$ . Determine the angular rate of rotation  $\dot{\theta}$  of the radial line  $r$  and the magnitude of the truck's acceleration.

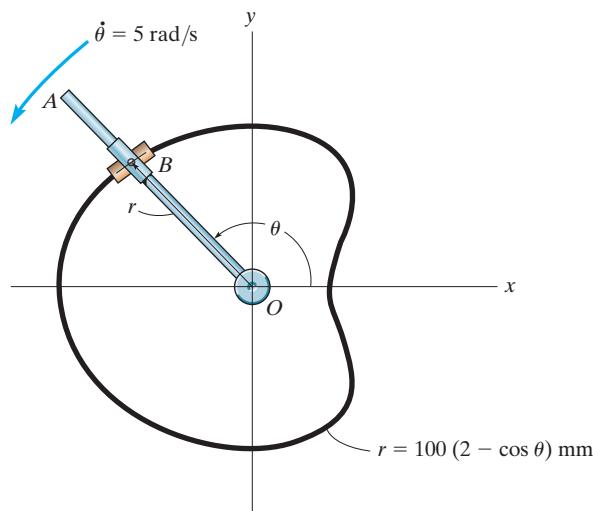
**\*12-184.** A truck is traveling along the horizontal circular curve of radius  $r = 60 \text{ m}$  with a speed of  $20 \text{ m/s}$  which is increasing at  $3 \text{ m/s}^2$ . Determine the truck's radial and transverse components of acceleration.



Probs. 12-183/184

**12-185.** The rod  $OA$  rotates counterclockwise with a constant angular velocity of  $\dot{\theta} = 5 \text{ rad/s}$ . Two pin-connected slider blocks, located at  $B$ , move freely on  $OA$  and the curved rod whose shape is a limaçon described by the equation  $r = 100(2 - \cos \theta) \text{ mm}$ . Determine the speed of the slider blocks at the instant  $\theta = 120^\circ$ .

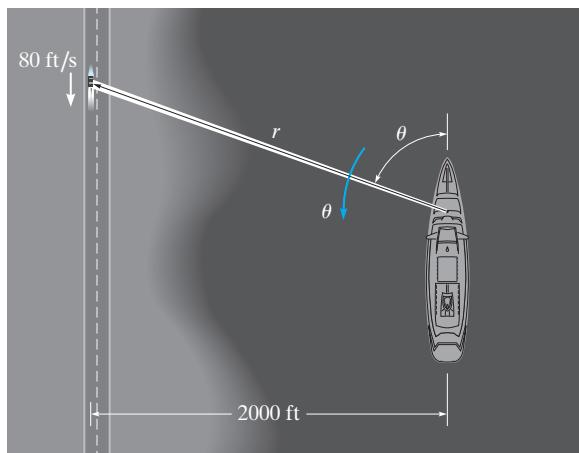
**12-186.** Determine the magnitude of the acceleration of the slider blocks in Prob. 12-185 when  $\theta = 120^\circ$ .



Probs. 12-185/186

**12-187.** The searchlight on the boat anchored 2000 ft from shore is turned on the automobile, which is traveling along the straight road at a constant speed 80 ft/s. Determine the angular rate of rotation of the light when the automobile is  $r = 3000$  ft from the boat.

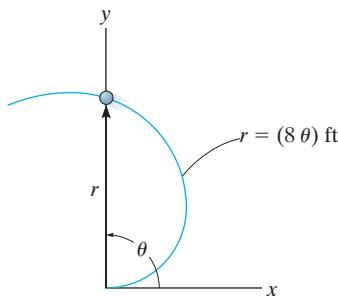
\***12-188.** If the car in Prob. 12-187 is accelerating at 15 ft/s<sup>2</sup> at the instant  $r = 3000$  ft determine the required angular acceleration  $\ddot{\theta}$  of the light at this instant.



Probs. 12-187/188

**12-189.** A particle moves along an Archimedean spiral  $r = (8\theta)$  ft, where  $\theta$  is given in radians. If  $\dot{\theta} = 4$  rad/s (constant), determine the radial and transverse components of the particle's velocity and acceleration at the instant  $\theta = \pi/2$  rad. Sketch the curve and show the components on the curve.

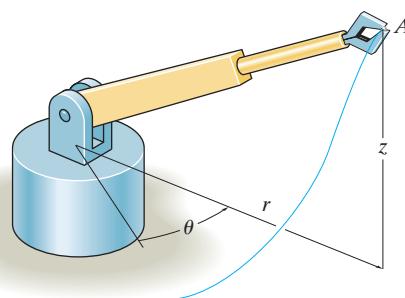
**12-190.** Solve Prob. 12-189 if the particle has an angular acceleration  $\ddot{\theta} = 5$  rad/s<sup>2</sup> when  $\dot{\theta} = 4$  rad/s at  $\theta = \pi/2$  rad.



Probs. 12-189/190

**12-191.** The arm of the robot moves so that  $r = 3$  ft is constant, and its grip  $A$  moves along the path  $z = (3 \sin 4\theta)$  ft, where  $\theta$  is in radians. If  $\theta = (0.5t)$  rad, where  $t$  is in seconds, determine the magnitudes of the grip's velocity and acceleration when  $t = 3$  s.

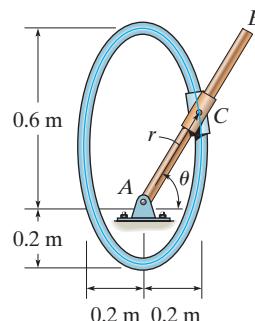
\***12-192.** For a short time the arm of the robot is extending such that  $\dot{r} = 1.5$  ft/s when  $r = 3$  ft,  $z = (4t^2)$  ft, and  $\theta = 0.5t$  rad, where  $t$  is in seconds. Determine the magnitudes of the velocity and acceleration of the grip  $A$  when  $t = 3$  s.



Probs. 12-191/192

**12-193.** The double collar  $C$  is pin connected together such that one collar slides over the fixed rod and the other slides over the rotating rod  $AB$ . If the angular velocity of  $AB$  is given as  $\dot{\theta} = (e^{0.5t^2})$  rad/s, where  $t$  is in seconds, and the path defined by the fixed rod is  $r = |(0.4 \sin \theta + 0.2)|$  m, determine the radial and transverse components of the collar's velocity and acceleration when  $t = 1$  s. When  $t = 0, \theta = 0$ . Use Simpson's rule with  $n = 50$  to determine  $\theta$  at  $t = 1$  s.

**12-194.** The double collar  $C$  is pin connected together such that one collar slides over the fixed rod and the other slides over the rotating rod  $AB$ . If the mechanism is to be designed so that the largest speed given to the collar is 6 m/s, determine the required constant angular velocity  $\dot{\theta}$  of rod  $AB$ . The path defined by the fixed rod is  $r = (0.4 \sin \theta + 0.2)$  m.



Probs. 12-193/194

## 12.9 Absolute Dependent Motion Analysis of Two Particles

In some types of problems the motion of one particle will *depend* on the corresponding motion of another particle. This dependency commonly occurs if the particles, here represented by blocks, are interconnected by inextensible cords which are wrapped around pulleys. For example, the movement of block *A* downward along the inclined plane in Fig. 12–36 will cause a corresponding movement of block *B* up the other incline. We can show this mathematically by first specifying the location of the blocks using *position coordinates*  $s_A$  and  $s_B$ . Note that each of the coordinate axes is (1) measured from a *fixed point* (*O*) or *fixed datum line*, (2) measured along each inclined plane *in the direction of motion* of each block, and (3) has a positive sense from the fixed datums to *A* and to *B*. If the total cord length is  $l_T$ , the two position coordinates are related by the equation

$$s_A + l_{CD} + s_B = l_T$$

Here  $l_{CD}$  is the length of the cord passing over arc *CD*. Taking the time derivative of this expression, realizing that  $l_{CD}$  and  $l_T$  remain constant, while  $s_A$  and  $s_B$  measure the segments of the cord that change in length, we have

$$\frac{ds_A}{dt} + \frac{ds_B}{dt} = 0 \quad \text{or} \quad v_B = -v_A$$

The negative sign indicates that when block *A* has a velocity downward, i.e., in the direction of positive  $s_A$ , it causes a corresponding upward velocity of block *B*; i.e., *B* moves in the negative  $s_B$  direction.

In a similar manner, time differentiation of the velocities yields the relation between the accelerations, i.e.,

$$a_B = -a_A$$

A more complicated example is shown in Fig. 12–37a. In this case, the position of block *A* is specified by  $s_A$ , and the position of the *end* of the cord from which block *B* is suspended is defined by  $s_B$ . As above, we have chosen position coordinates which (1) have their origin at fixed points or datums, (2) are measured in the direction of motion of each block, and (3) from the fixed datums are positive to the right for  $s_A$  and positive downward for  $s_B$ . During the motion, the length of the red colored segments of the cord in Fig. 12–37a remains constant. If  $l$  represents the total length of cord minus these segments, then the position coordinates can be related by the equation

$$2s_B + h + s_A = l$$

Since  $l$  and  $h$  are constant during the motion, the two time derivatives yield

$$2v_B = -v_A \quad 2a_B = -a_A$$

Hence, when *B* moves downward ( $+s_B$ ), *A* moves to the left ( $-s_A$ ) with twice the motion.

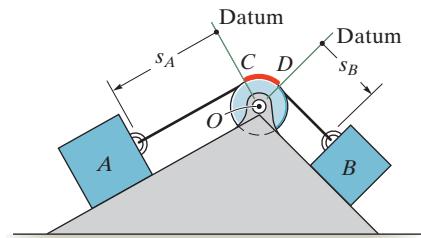


Fig. 12–36

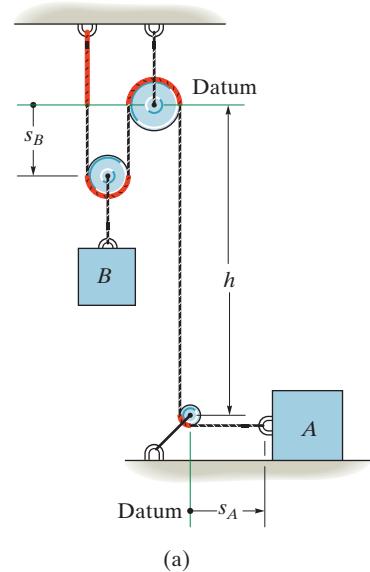
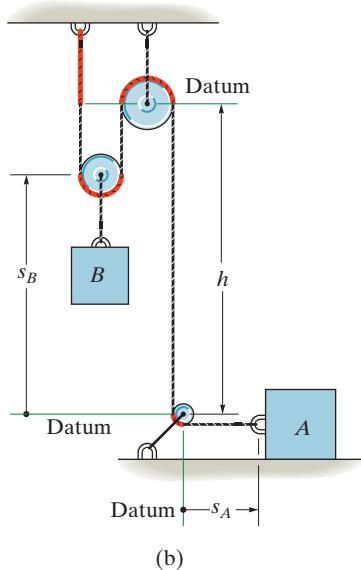


Fig. 12–37



**Fig. 12-37 (cont.)**

This example can also be worked by defining the position of block *B* from the center of the bottom pulley (a fixed point), Fig. 12-37*b*. In this case

$$2(h - s_B) + h + s_A = l$$

Time differentiation yields

$$2v_B = v_A \quad 2a_B = a_A$$

Here the signs are the same. Why?

### Procedure for Analysis

The above method of relating the dependent motion of one particle to that of another can be performed using algebraic scalars or position coordinates provided each particle moves along a rectilinear path. When this is the case, only the magnitudes of the velocity and acceleration of the particles will change, not their line of direction.

#### Position-Coordinate Equation.

- Establish each position coordinate with an origin located at a *fixed* point or datum.
- It is *not necessary* that the *origin* be the *same* for each of the coordinates; however, it is *important* that each coordinate axis selected be directed along the *path of motion* of the particle.
- Using geometry or trigonometry, relate the position coordinates to the total length of the cord,  $l_T$ , or to that portion of cord,  $l$ , which *excludes* the segments that do not change length as the particles move—such as arc segments wrapped over pulleys.
- If a problem involves a *system* of two or more cords wrapped around pulleys, then the position of a point on one cord must be related to the position of a point on another cord using the above procedure. Separate equations are written for a fixed length of each cord of the system and the positions of the two particles are then related by these equations (see Examples 12.22 and 12.23).

#### Time Derivatives.

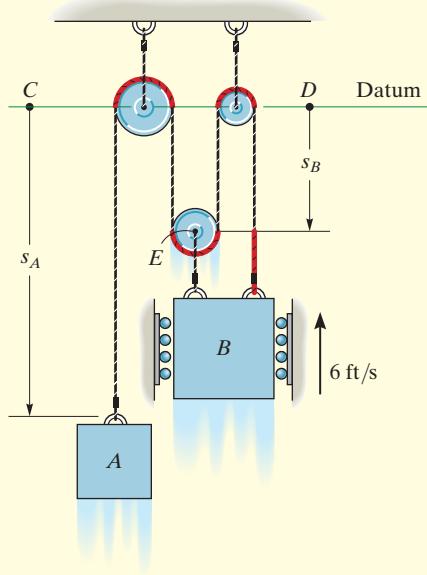
- Two successive time derivatives of the position-coordinate equations yield the required velocity and acceleration equations which relate the motions of the particles.
- The signs of the terms in these equations will be consistent with those that specify the positive and negative sense of the position coordinates.



The cable is wrapped around the pulleys on this crane in order to reduce the required force needed to hoist a load. (© R.C. Hibbeler)

**EXAMPLE | 12.21**

Determine the speed of block A in Fig. 12–38 if block B has an upward speed of 6 ft/s.



**Fig. 12–38**

**SOLUTION**

**Position-Coordinate Equation.** There is *one cord* in this system having segments which change length. Position coordinates  $s_A$  and  $s_B$  will be used since each is measured from a fixed point (C or D) and extends along each block's *path of motion*. In particular,  $s_B$  is directed to point E since motion of B and E is the same.

The red colored segments of the cord in Fig. 12–38 remain at a constant length and do not have to be considered as the blocks move. The remaining length of cord,  $l$ , is also constant and is related to the changing position coordinates  $s_A$  and  $s_B$  by the equation

$$s_A + 3s_B = l$$

**Time Derivative.** Taking the time derivative yields

$$v_A + 3v_B = 0$$

so that when  $v_B = -6 \text{ ft/s}$  (upward),

$$v_A = 18 \text{ ft/s} \downarrow$$

*Ans.*

## EXAMPLE 12.22

Determine the speed of  $A$  in Fig. 12–39 if  $B$  has an upward speed of 6 ft/s.

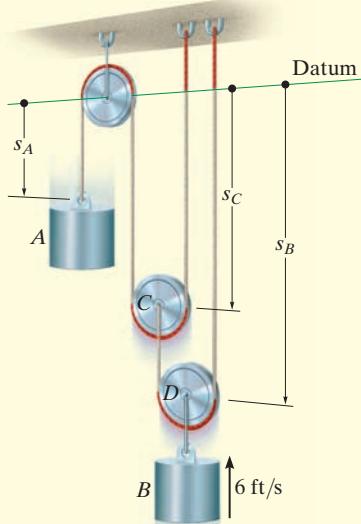


Fig. 12–39

### SOLUTION

**Position-Coordinate Equation.** As shown, the positions of blocks  $A$  and  $B$  are defined using coordinates  $s_A$  and  $s_B$ . Since the system has two cords with segments that change length, it will be necessary to use a third coordinate,  $s_C$ , in order to relate  $s_A$  to  $s_B$ . In other words, the length of one of the cords can be expressed in terms of  $s_A$  and  $s_C$ , and the length of the other cord can be expressed in terms of  $s_B$  and  $s_C$ .

The red colored segments of the cords in Fig. 12–39 do not have to be considered in the analysis. Why? For the remaining cord lengths, say  $l_1$  and  $l_2$ , we have

$$s_A + 2s_C = l_1 \quad s_B + (s_B - s_C) = l_2$$

**Time Derivative.** Taking the time derivative of these equations yields

$$v_A + 2v_C = 0 \quad 2v_B - v_C = 0$$

Eliminating  $v_C$  produces the relationship between the motions of each cylinder.

$$v_A + 4v_B = 0$$

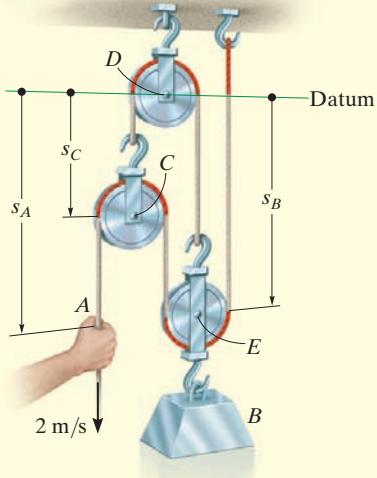
so that when  $v_B = -6$  ft/s (upward),

$$v_A = +24 \text{ ft/s} = 24 \text{ ft/s} \downarrow$$

*Ans.*

**EXAMPLE | 12.23**

Determine the speed of block *B* in Fig. 12–40 if the end of the cord at *A* is pulled down with a speed of 2 m/s.



**Fig. 12–40**

**SOLUTION**

**Position-Coordinate Equation.** The position of point *A* is defined by  $s_A$ , and the position of block *B* is specified by  $s_B$  since point *E* on the pulley will have the *same motion* as the block. Both coordinates are measured from a horizontal datum passing through the *fixed pin* at pulley *D*. Since the system consists of *two* cords, the coordinates  $s_A$  and  $s_B$  cannot be related directly. Instead, by establishing a third position coordinate,  $s_C$ , we can now express the length of one of the cords in terms of  $s_B$  and  $s_C$ , and the length of the other cord in terms of  $s_A$ ,  $s_B$ , and  $s_C$ .

Excluding the red colored segments of the cords in Fig. 12–40, the remaining constant cord lengths  $l_1$  and  $l_2$  (along with the hook and link dimensions) can be expressed as

$$\begin{aligned} s_C + s_B &= l_1 \\ (s_A - s_C) + (s_B - s_C) + s_B &= l_2 \end{aligned}$$

**Time Derivative.** The time derivative of each equation gives

$$\begin{aligned} v_C + v_B &= 0 \\ v_A - 2v_C + 2v_B &= 0 \end{aligned}$$

Eliminating  $v_C$ , we obtain

$$v_A + 4v_B = 0$$

so that when  $v_A = 2 \text{ m/s}$  (downward),

$$v_B = -0.5 \text{ m/s} = 0.5 \text{ m/s} \uparrow \quad \text{Ans.}$$

## EXAMPLE 12.24

A man at  $A$  is hoisting a safe  $S$  as shown in Fig. 12–41 by walking to the right with a constant velocity  $v_A = 0.5 \text{ m/s}$ . Determine the velocity and acceleration of the safe when it reaches the elevation of 10 m. The rope is 30 m long and passes over a small pulley at  $D$ .

## SOLUTION

**Position-Coordinate Equation.** This problem is unlike the previous examples since rope segment  $DA$  changes *both direction and magnitude*. However, the ends of the rope, which define the positions of  $C$  and  $A$ , are specified by means of the  $x$  and  $y$  coordinates since they must be measured from a fixed point and *directed along the paths of motion* of the ends of the rope.

The  $x$  and  $y$  coordinates may be related since the rope has a fixed length  $l = 30 \text{ m}$ , which at all times is equal to the length of segment  $DA$  plus  $CD$ . Using the Pythagorean theorem to determine  $l_{DA}$ , we have  $l_{DA} = \sqrt{(15)^2 + x^2}$ ; also,  $l_{CD} = 15 - y$ . Hence,

$$\begin{aligned} l &= l_{DA} + l_{CD} \\ 30 &= \sqrt{(15)^2 + x^2} + (15 - y) \\ y &= \sqrt{225 + x^2} - 15 \end{aligned} \quad (1)$$

**Time Derivatives.** Taking the time derivative, using the chain rule (see Appendix C), where  $v_S = dy/dt$  and  $v_A = dx/dt$ , yields

$$\begin{aligned} v_S &= \frac{dy}{dt} = \left[ \frac{1}{2} \frac{2x}{\sqrt{225 + x^2}} \right] \frac{dx}{dt} \\ &= \frac{x}{\sqrt{225 + x^2}} v_A \end{aligned} \quad (2)$$

At  $y = 10 \text{ m}$ ,  $x$  is determined from Eq. 1, i.e.,  $x = 20 \text{ m}$ . Hence, from Eq. 2 with  $v_A = 0.5 \text{ m/s}$ ,

$$v_S = \frac{20}{\sqrt{225 + (20)^2}} (0.5) = 0.4 \text{ m/s} = 400 \text{ mm/s} \uparrow \quad \text{Ans.}$$

The acceleration is determined by taking the time derivative of Eq. 2. Since  $v_A$  is constant, then  $a_A = dv_A/dt = 0$ , and we have

$$a_S = \frac{d^2y}{dt^2} = \left[ \frac{-x(dx/dt)}{(225 + x^2)^{3/2}} \right] xv_A + \left[ \frac{1}{\sqrt{225 + x^2}} \right] \left( \frac{dx}{dt} \right) v_A + \left[ \frac{1}{\sqrt{225 + x^2}} \right] x \frac{dv_A}{dt} = \frac{225v_A^2}{(225 + x^2)^{3/2}}$$

At  $x = 20 \text{ m}$ , with  $v_A = 0.5 \text{ m/s}$ , the acceleration becomes

$$a_S = \frac{225(0.5 \text{ m/s})^2}{[225 + (20 \text{ m})^2]^{3/2}} = 0.00360 \text{ m/s}^2 = 3.60 \text{ mm/s}^2 \uparrow \quad \text{Ans.}$$

**NOTE:** The constant velocity at  $A$  causes the other end  $C$  of the rope to have an acceleration since  $v_A$  causes segment  $DA$  to change its direction as well as its length.

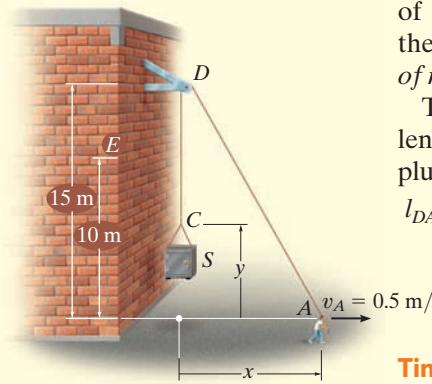


Fig. 12–41

## 12.10 Relative-Motion of Two Particles Using Translating Axes

Throughout this chapter the absolute motion of a particle has been determined using a single fixed reference frame. There are many cases, however, where the path of motion for a particle is complicated, so that it may be easier to analyze the motion in parts by using two or more frames of reference. For example, the motion of a particle located at the tip of an airplane propeller, while the plane is in flight, is more easily described if one observes first the motion of the airplane from a fixed reference and then superimposes (vectorially) the circular motion of the particle measured from a reference attached to the airplane.

In this section *translating frames of reference* will be considered for the analysis.

**Position.** Consider particles *A* and *B*, which move along the arbitrary paths shown in Fig. 12-42. The *absolute position* of each particle,  $\mathbf{r}_A$  and  $\mathbf{r}_B$ , is measured from the common origin *O* of the *fixed*  $x$ ,  $y$ ,  $z$  reference frame. The origin of a second frame of reference  $x'$ ,  $y'$ ,  $z'$  is attached to and moves with particle *A*. The axes of this frame are *only permitted to translate* relative to the fixed frame. The position of *B* measured relative to *A* is denoted by the *relative-position vector*  $\mathbf{r}_{B/A}$ . Using vector addition, the three vectors shown in Fig. 12-42 can be related by the equation

$$\mathbf{r}_B = \mathbf{r}_A + \mathbf{r}_{B/A} \quad (12-33)$$

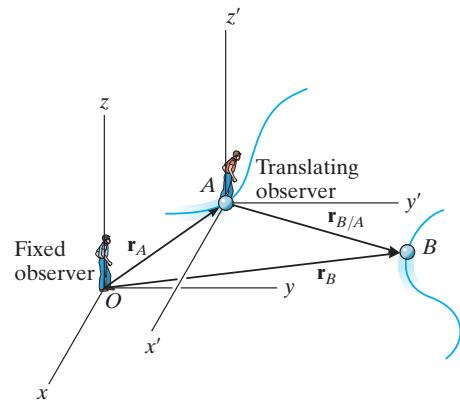


Fig. 12-42

**Velocity.** An equation that relates the velocities of the particles is determined by taking the time derivative of the above equation; i.e.,

$$\mathbf{v}_B = \mathbf{v}_A + \mathbf{v}_{B/A} \quad (12-34)$$

Here  $\mathbf{v}_B = d\mathbf{r}_B/dt$  and  $\mathbf{v}_A = d\mathbf{r}_A/dt$  refer to *absolute velocities*, since they are observed from the fixed frame; whereas the *relative velocity*  $\mathbf{v}_{B/A} = d\mathbf{r}_{B/A}/dt$  is observed from the translating frame. It is important to note that since the  $x'$ ,  $y'$ ,  $z'$  axes translate, the *components* of  $\mathbf{r}_{B/A}$  will *not* change direction and therefore the time derivative of these components will only have to account for the change in their magnitudes. Equation 12-34 therefore states that the velocity of *B* is equal to the velocity of *A* plus (vectorially) the velocity of "B with respect to A," as measured by the *translating observer* fixed in the  $x'$ ,  $y'$ ,  $z'$  reference frame.

**Acceleration.** The time derivative of Eq. 12–34 yields a similar vector relation between the *absolute* and *relative accelerations* of particles *A* and *B*.

$$\mathbf{a}_B = \mathbf{a}_A + \mathbf{a}_{B/A} \quad (12-35)$$

Here  $\mathbf{a}_{B/A}$  is the acceleration of *B* as seen by the observer located at *A* and translating with the  $x'$ ,  $y'$ ,  $z'$  reference frame.\*

### Procedure for Analysis

- When applying the relative velocity and acceleration equations, it is first necessary to specify the particle *A* that is the origin for the translating  $x'$ ,  $y'$ ,  $z'$  axes. Usually this point has a *known* velocity or acceleration.
- Since vector addition forms a triangle, there can be at most *two unknowns*, represented by the magnitudes and/or directions of the vector quantities.
- These unknowns can be solved for either graphically, using trigonometry (law of sines, law of cosines), or by resolving each of the three vectors into rectangular or Cartesian components, thereby generating a set of scalar equations.



The pilots of these close-flying planes must be aware of their relative positions and velocities at all times in order to avoid a collision. (© R.C. Hibbeler)

\* An easy way to remember the setup of these equations is to note the “cancellation” of the subscript *A* between the two terms, e.g.,  $\mathbf{a}_B = \mathbf{a}_A + \mathbf{a}_{B/A}$ .

**EXAMPLE | 12.25**

A train travels at a constant speed of 60 mi/h and crosses over a road as shown in Fig. 12–43a. If the automobile  $A$  is traveling at 45 mi/h along the road, determine the magnitude and direction of the velocity of the train relative to the automobile.

**SOLUTION I**

**Vector Analysis.** The relative velocity  $\mathbf{v}_{T/A}$  is measured from the translating  $x'$ ,  $y'$  axes attached to the automobile, Fig. 12–43a. It is determined from  $\mathbf{v}_T = \mathbf{v}_A + \mathbf{v}_{T/A}$ . Since  $\mathbf{v}_T$  and  $\mathbf{v}_A$  are known in both magnitude and direction, the unknowns become the  $x$  and  $y$  components of  $\mathbf{v}_{T/A}$ . Using the  $x$ ,  $y$  axes in Fig. 12–43a, we have

$$\mathbf{v}_T = \mathbf{v}_A + \mathbf{v}_{T/A}$$

$$60\mathbf{i} = (45 \cos 45^\circ\mathbf{i} + 45 \sin 45^\circ\mathbf{j}) + \mathbf{v}_{T/A}$$

$$\mathbf{v}_{T/A} = \{28.2\mathbf{i} - 31.8\mathbf{j}\} \text{ mi/h}$$

The magnitude of  $\mathbf{v}_{T/A}$  is thus

$$v_{T/A} = \sqrt{(28.2)^2 + (-31.8)^2} = 42.5 \text{ mi/h} \quad \text{Ans.}$$

From the direction of each component, Fig. 12–43b, the direction of  $\mathbf{v}_{T/A}$  is

$$\tan \theta = \frac{(v_{T/A})_y}{(v_{T/A})_x} = \frac{31.8}{28.2}$$

$$\theta = 48.5^\circ \quad \text{Ans.}$$

Note that the vector addition shown in Fig. 12–43b indicates the correct sense for  $\mathbf{v}_{T/A}$ . This figure anticipates the answer and can be used to check it.

**SOLUTION II**

**Scalar Analysis.** The unknown components of  $\mathbf{v}_{T/A}$  can also be determined by applying a scalar analysis. We will assume these components act in the *positive*  $x$  and  $y$  directions. Thus,

$$\mathbf{v}_T = \mathbf{v}_A + \mathbf{v}_{T/A}$$

$$\left[ \begin{array}{c} 60 \text{ mi/h} \\ \rightarrow \end{array} \right] = \left[ \begin{array}{c} 45 \text{ mi/h} \\ \nearrow 45^\circ \end{array} \right] + \left[ \begin{array}{c} (v_{T/A})_x \\ \rightarrow \end{array} \right] + \left[ \begin{array}{c} (v_{T/A})_y \\ \uparrow \end{array} \right]$$

Resolving each vector into its  $x$  and  $y$  components yields

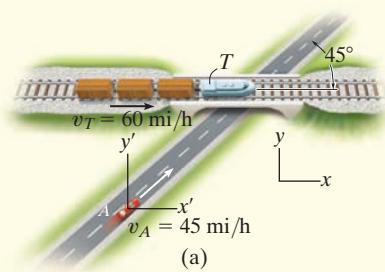
$$(\pm) \quad 60 = 45 \cos 45^\circ + (v_{T/A})_x + 0$$

$$(+\uparrow) \quad 0 = 45 \sin 45^\circ + 0 + (v_{T/A})_y$$

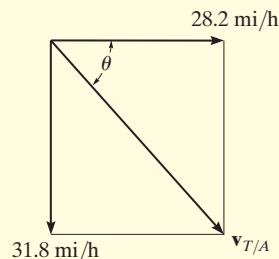
Solving, we obtain the previous results,

$$(v_{T/A})_x = 28.2 \text{ mi/h} = 28.2 \text{ mi/h} \rightarrow$$

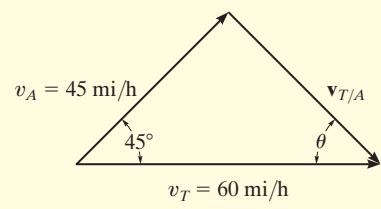
$$(v_{T/A})_y = -31.8 \text{ mi/h} = 31.8 \text{ mi/h} \downarrow$$



(a)



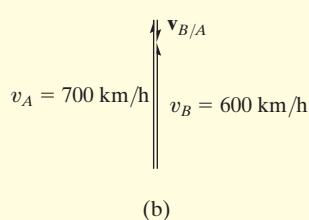
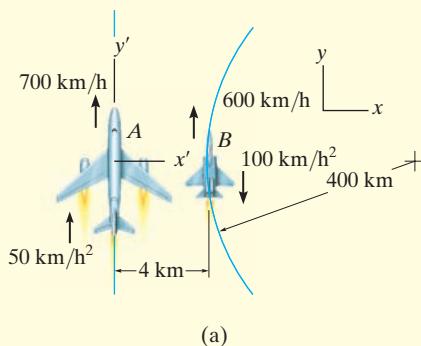
(b)



(c)

**Fig. 12–43**

## EXAMPLE 12.26



Plane A in Fig. 12–44a is flying along a straight-line path, whereas plane B is flying along a circular path having a radius of curvature of  $\rho_B = 400 \text{ km}$ . Determine the velocity and acceleration of B as measured by the pilot of A.

## SOLUTION

**Velocity.** The origin of the  $x$  and  $y$  axes are located at an arbitrary fixed point. Since the motion relative to plane A is to be determined, the *translating frame of reference*  $x'$ ,  $y'$  is attached to it, Fig. 12–44a. Applying the relative-velocity equation in scalar form since the velocity vectors of both planes are parallel at the instant shown, we have

$$(+\uparrow) \quad v_B = v_A + v_{B/A}$$

$$600 \text{ km/h} = 700 \text{ km/h} + v_{B/A}$$

$$v_{B/A} = -100 \text{ km/h} = 100 \text{ km/h} \downarrow \quad \text{Ans.}$$

The vector addition is shown in Fig. 12–44b.

**Acceleration.** Plane B has both tangential and normal components of acceleration since it is flying along a *curved path*. From Eq. 12–20, the magnitude of the normal component is

$$(a_B)_n = \frac{v_B^2}{\rho} = \frac{(600 \text{ km/h})^2}{400 \text{ km}} = 900 \text{ km/h}^2$$

Applying the relative-acceleration equation gives

$$\mathbf{a}_B = \mathbf{a}_A + \mathbf{a}_{B/A}$$

$$900\mathbf{i} - 100\mathbf{j} = 50\mathbf{j} + \mathbf{a}_{B/A}$$

Thus,

$$\mathbf{a}_{B/A} = \{900\mathbf{i} - 150\mathbf{j}\} \text{ km/h}^2$$

From Fig. 12–44c, the magnitude and direction of  $\mathbf{a}_{B/A}$  are therefore

$$a_{B/A} = 912 \text{ km/h}^2 \quad \theta = \tan^{-1} \frac{150}{900} = 9.46^\circ \quad \text{Ans.}$$

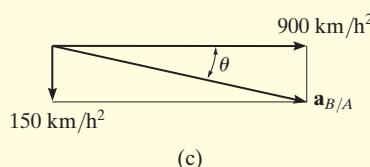


Fig. 12–44

**NOTE:** The solution to this problem was possible using a translating frame of reference, since the pilot in plane A is “translating.” Observation of the motion of plane A with respect to the pilot of plane B, however, must be obtained using a *rotating set of axes* attached to plane B. (This assumes, of course, that the pilot of B is fixed in the rotating frame, so he does not turn his eyes to follow the motion of A.) The analysis for this case is given in Example 16.21.

**EXAMPLE | 12.27**

At the instant shown in Fig. 12–45a, cars A and B are traveling with speeds of 18 m/s and 12 m/s, respectively. Also at this instant, A has a decrease in speed of  $2 \text{ m/s}^2$ , and B has an increase in speed of  $3 \text{ m/s}^2$ . Determine the velocity and acceleration of B with respect to A.

**SOLUTION**

**Velocity.** The fixed  $x, y$  axes are established at an arbitrary point on the ground and the translating  $x', y'$  axes are attached to car A, Fig. 12–45a. Why? The relative velocity is determined from  $\mathbf{v}_B = \mathbf{v}_A + \mathbf{v}_{B/A}$ . What are the two unknowns? Using a Cartesian vector analysis, we have

$$\begin{aligned}\mathbf{v}_B &= \mathbf{v}_A + \mathbf{v}_{B/A} \\ -12\mathbf{j} &= (-18 \cos 60^\circ \mathbf{i} - 18 \sin 60^\circ \mathbf{j}) + \mathbf{v}_{B/A} \\ \mathbf{v}_{B/A} &= \{9\mathbf{i} + 3.588\mathbf{j}\} \text{ m/s}\end{aligned}$$

Thus,

$$v_{B/A} = \sqrt{(9)^2 + (3.588)^2} = 9.69 \text{ m/s} \quad \text{Ans.}$$

Noting that  $\mathbf{v}_{B/A}$  has  $+\mathbf{i}$  and  $+\mathbf{j}$  components, Fig. 12–45b, its direction is

$$\begin{aligned}\tan \theta &= \frac{(v_{B/A})_y}{(v_{B/A})_x} = \frac{3.588}{9} \\ \theta &= 21.7^\circ \swarrow \quad \text{Ans.}\end{aligned}$$

**Acceleration.** Car B has both tangential and normal components of acceleration. Why? The magnitude of the normal component is

$$(a_B)_n = \frac{v_B^2}{\rho} = \frac{(12 \text{ m/s})^2}{100 \text{ m}} = 1.440 \text{ m/s}^2$$

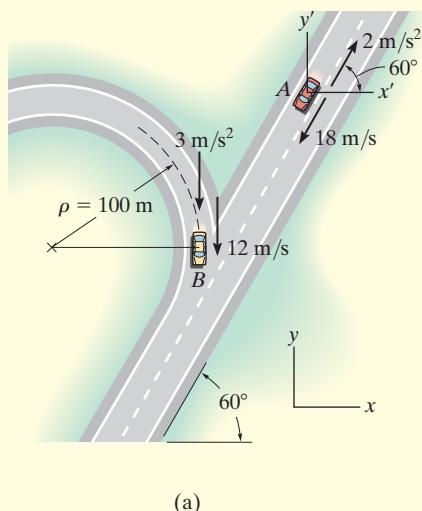
Applying the equation for relative acceleration yields

$$\begin{aligned}\mathbf{a}_B &= \mathbf{a}_A + \mathbf{a}_{B/A} \\ (-1.440\mathbf{i} - 3\mathbf{j}) &= (2 \cos 60^\circ \mathbf{i} + 2 \sin 60^\circ \mathbf{j}) + \mathbf{a}_{B/A} \\ \mathbf{a}_{B/A} &= \{-2.440\mathbf{i} - 4.732\mathbf{j}\} \text{ m/s}^2\end{aligned}$$

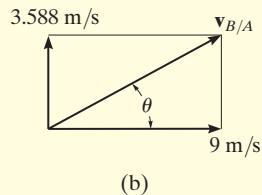
Here  $\mathbf{a}_{B/A}$  has  $-\mathbf{i}$  and  $-\mathbf{j}$  components. Thus, from Fig. 12–45c,

$$\begin{aligned}a_{B/A} &= \sqrt{(2.440)^2 + (4.732)^2} = 5.32 \text{ m/s}^2 \quad \text{Ans.} \\ \tan \phi &= \frac{(a_{B/A})_y}{(a_{B/A})_x} = \frac{4.732}{2.440} \\ \phi &= 62.7^\circ \nwarrow \quad \text{Ans.}\end{aligned}$$

**NOTE:** Is it possible to obtain the relative acceleration of  $\mathbf{a}_{A/B}$  using this method? Refer to the comment made at the end of Example 12.26.



(a)



(b)

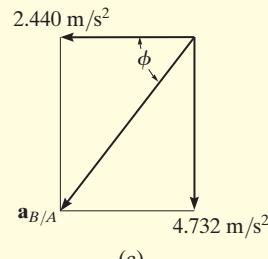
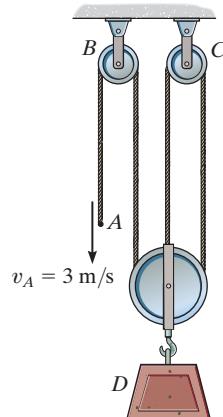


Fig. 12–45

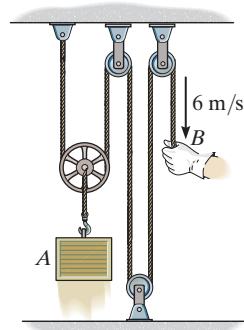
## FUNDAMENTAL PROBLEMS

**F12–39.** Determine the velocity of block *D* if end *A* of the rope is pulled down with a speed of  $v_A = 3 \text{ m/s}$ .



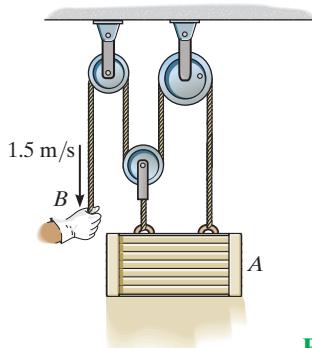
Prob. F12–39

**F12–40.** Determine the velocity of block *A* if end *B* of the rope is pulled down with a speed of  $6 \text{ m/s}$ .



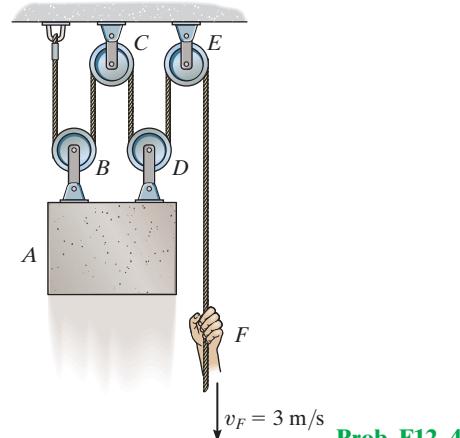
Prob. F12–40

**F12–41.** Determine the velocity of block *A* if end *B* of the rope is pulled down with a speed of  $1.5 \text{ m/s}$ .



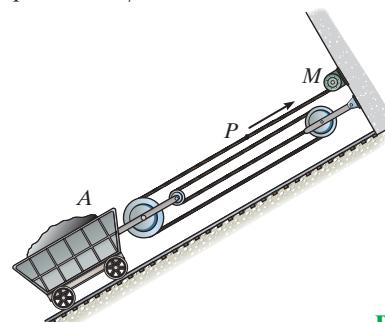
Prob. F12–41

**F12–42.** Determine the velocity of block *A* if end *F* of the rope is pulled down with a speed of  $v_F = 3 \text{ m/s}$ .



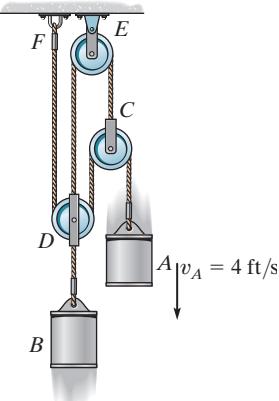
Prob. F12–42

**F12–43.** Determine the velocity of car *A* if point *P* on the cable has a speed of  $4 \text{ m/s}$  when the motor *M* winds the cable in.



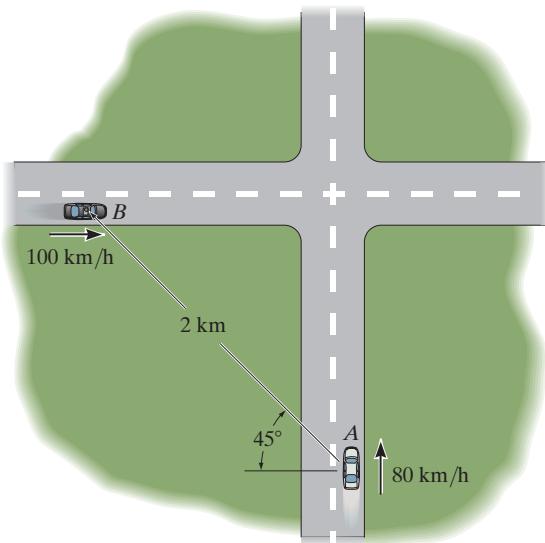
Prob. F12–43

**F12–44.** Determine the velocity of cylinder *B* if cylinder *A* moves downward with a speed of  $v_A = 4 \text{ ft/s}$ .



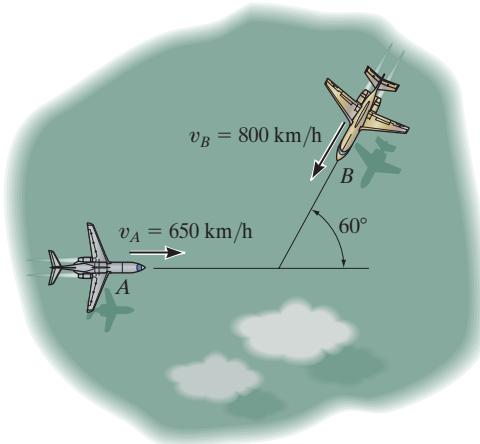
Prob. F12–44

**F12-45.** Car *A* is traveling with a constant speed of 80 km/h due north, while car *B* is traveling with a constant speed of 100 km/h due east. Determine the velocity of car *B* relative to car *A*.



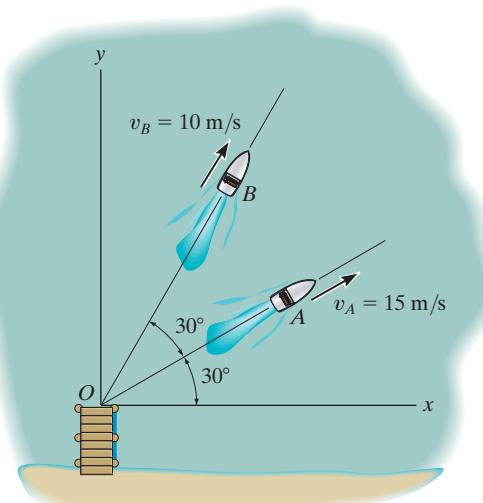
Prob. F12-45

**F12-46.** Two planes *A* and *B* are traveling with the constant velocities shown. Determine the magnitude and direction of the velocity of plane *B* relative to plane *A*.



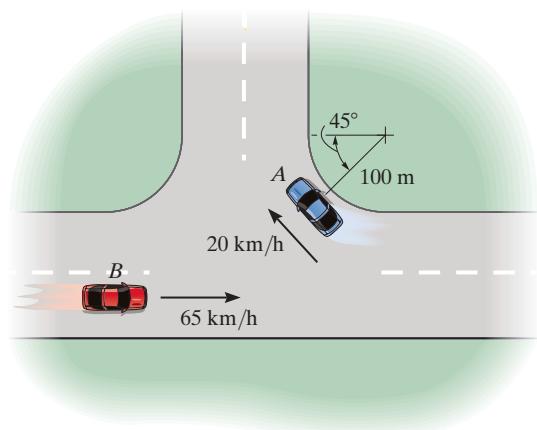
Prob. F12-46

**F12-47.** The boats *A* and *B* travel with constant speeds of  $v_A = 15 \text{ m/s}$  and  $v_B = 10 \text{ m/s}$  when they leave the pier at *O* at the same time. Determine the distance between them when  $t = 4 \text{ s}$ .



Prob. F12-47

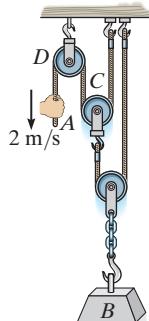
**F12-48.** At the instant shown, cars *A* and *B* are traveling at the speeds shown. If *B* is accelerating at  $1200 \text{ km/h}^2$  while *A* maintains a constant speed, determine the velocity and acceleration of *A* with respect to *B*.



Prob. F12-48

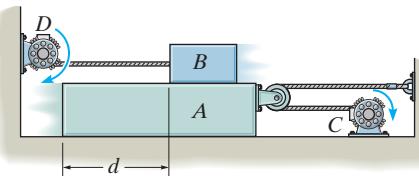
## PROBLEMS

**12-195.** If the end of the cable at *A* is pulled down with a speed of 2 m/s, determine the speed at which block *B* rises.



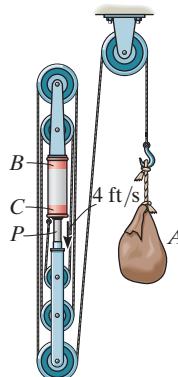
Prob. 12-195

\***12-196.** The motor at *C* pulls in the cable with an acceleration  $a_C = (3t^2) \text{ m/s}^2$ , where  $t$  is in seconds. The motor at *D* draws in its cable at  $a_D = 5 \text{ m/s}^2$ . If both motors start at the same instant from rest when  $d = 3 \text{ m}$ , determine (a) the time needed for  $d = 0$ , and (b) the velocities of blocks *A* and *B* when this occurs.



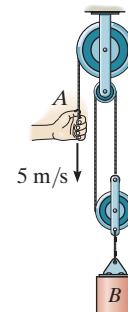
Prob. 12-196

**12-197.** The pulley arrangement shown is designed for hoisting materials. If *BC* remains fixed while the plunger *P* is pushed downward with a speed of 4 ft/s, determine the speed of the load at *A*.



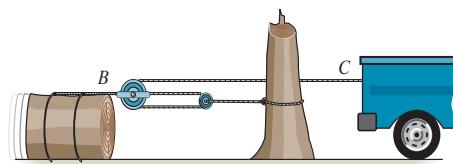
Prob. 12-197

**12-198.** If the end of the cable at *A* is pulled down with a speed of 5 m/s, determine the speed at which block *B* rises.



Prob. 12-198

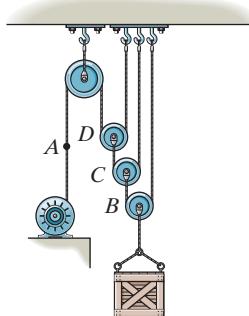
**12-199.** Determine the displacement of the log if the truck at *C* pulls the cable 4 ft to the right.



Prob. 12-199

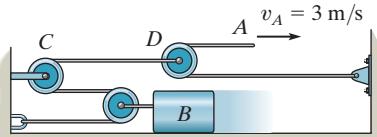
\***12-200.** Determine the constant speed at which the cable at *A* must be drawn in by the motor in order to hoist the load 6 m in 1.5 s.

**12-201.** Starting from rest, the cable can be wound onto the drum of the motor at a rate of  $v_A = (3t^2) \text{ m/s}$ , where  $t$  is in seconds. Determine the time needed to lift the load 7 m.



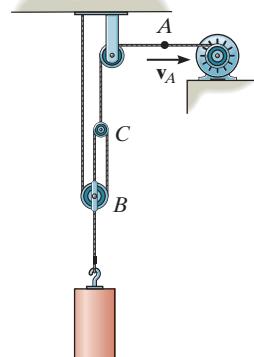
Probs. 12-200/201

- 12-202.** If the end *A* of the cable is moving at  $v_A = 3 \text{ m/s}$ , determine the speed of block *B*.

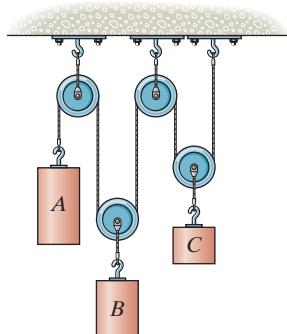
**Prob. 12-202**

- 12-203.** Determine the time needed for the load at *B* to attain a speed of  $10 \text{ m/s}$ , starting from rest, if the cable is drawn into the motor with an acceleration of  $3 \text{ m/s}^2$ .

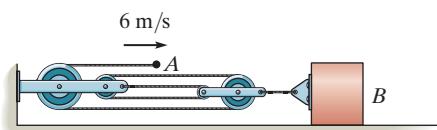
- \*12-204.** The cable at *A* is being drawn toward the motor at  $v_A = 8 \text{ m/s}$ . Determine the velocity of the block.

**Probs. 12-203/204**

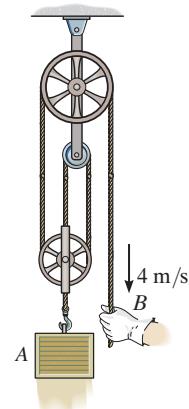
- 12-205.** If block *A* of the pulley system is moving downward at  $6 \text{ ft/s}$  while block *C* is moving down at  $18 \text{ ft/s}$ , determine the relative velocity of block *B* with respect to *C*.

**Prob. 12-205**

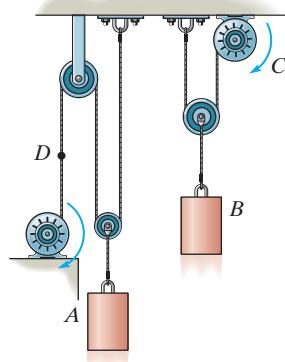
- 12-206.** Determine the speed of the block at *B*.

**Prob. 12-206**

- 12-207.** Determine the speed of block *A* if the end of the rope is pulled down with a speed of  $4 \text{ m/s}$ .

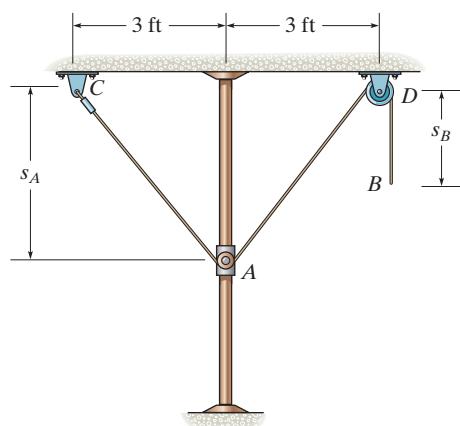
**Prob. 12-207**

- \*12-208.** The motor draws in the cable at *C* with a constant velocity of  $v_C = 4 \text{ m/s}$ . The motor draws in the cable at *D* with a constant acceleration of  $a_D = 8 \text{ m/s}^2$ . If  $v_D = 0$  when  $t = 0$ , determine (a) the time needed for block *A* to rise  $3 \text{ m}$ , and (b) the relative velocity of block *A* with respect to block *B* when this occurs.

**Prob. 12-208**

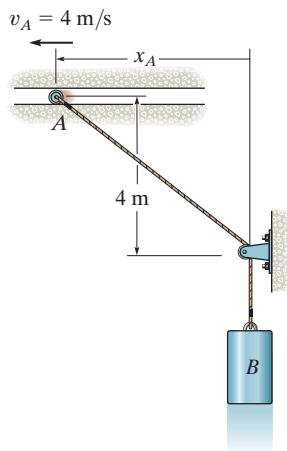
**12-209.** The cord is attached to the pin at *C* and passes over the two pulleys at *A* and *D*. The pulley at *A* is attached to the smooth collar that travels along the vertical rod. Determine the velocity and acceleration of the end of the cord at *B* if at the instant  $s_A = 4$  ft the collar is moving upward at 5 ft/s, which is decreasing at 2 ft/s<sup>2</sup>.

**12-210.** The 16-ft-long cord is attached to the pin at *C* and passes over the two pulleys at *A* and *D*. The pulley at *A* is attached to the smooth collar that travels along the vertical rod. When  $s_B = 6$  ft, the end of the cord at *B* is pulled downward with a velocity of 4 ft/s and is given an acceleration of 3 ft/s<sup>2</sup>. Determine the velocity and acceleration of the collar at this instant.



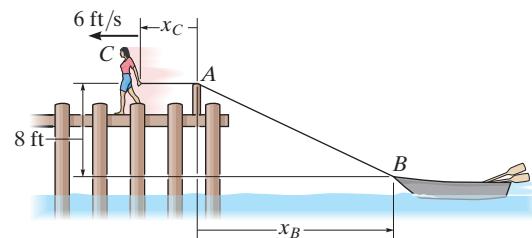
Probs. 12-209/210

**12-211.** The roller at *A* is moving with a velocity of  $v_A = 4$  m/s and has an acceleration of  $a_A = 2$  m/s<sup>2</sup> when  $x_A = 3$  m. Determine the velocity and acceleration of block *B* at this instant.



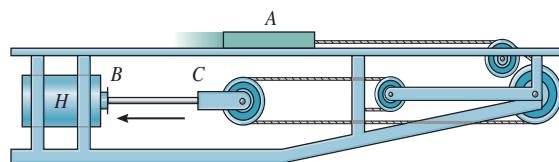
Prob. 12-211

**\*12-212.** The girl at *C* stands near the edge of the pier and pulls in the rope *horizontally* at a constant speed of 6 ft/s. Determine how fast the boat approaches the pier at the instant the rope length *AB* is 50 ft.



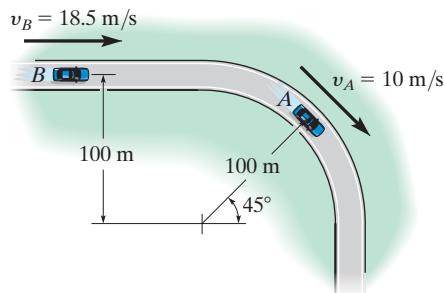
Prob. 12-212

**12-213.** If the hydraulic cylinder *H* draws in rod *BC* at 2 ft/s, determine the speed of slider *A*.



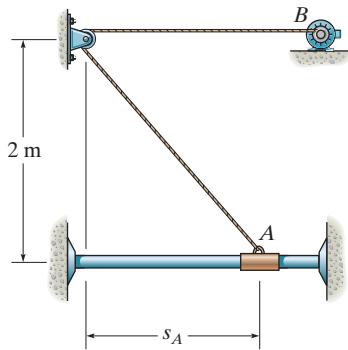
Prob. 12-213

**12-214.** At the instant shown, the car at *A* is traveling at 10 m/s around the curve while increasing its speed at 5 m/s<sup>2</sup>. The car at *B* is traveling at 18.5 m/s along the straightaway and increasing its speed at 2 m/s<sup>2</sup>. Determine the relative velocity and relative acceleration of *A* with respect to *B* at this instant.

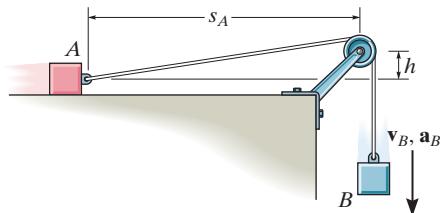


Prob. 12-214

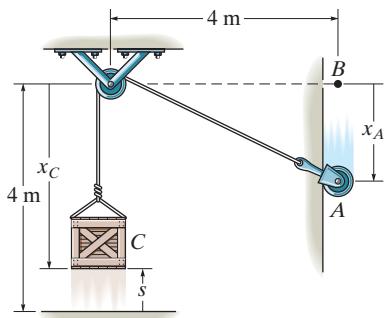
- 12-215.** The motor draws in the cord at  $B$  with an acceleration of  $a_B = 2 \text{ m/s}^2$ . When  $s_A = 1.5 \text{ m}$ ,  $v_B = 6 \text{ m/s}$ . Determine the velocity and acceleration of the collar at this instant.

**Prob. 12-215**

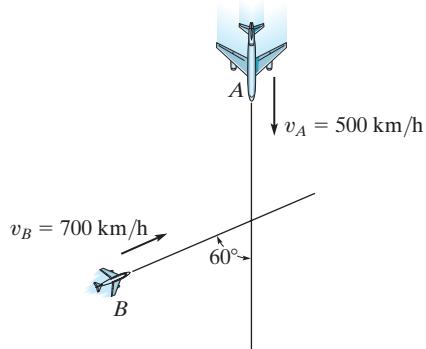
- \***12-216.** If block  $B$  is moving down with a velocity  $v_B$  and has an acceleration  $a_B$ , determine the velocity and acceleration of block  $A$  in terms of the parameters shown.

**Prob. 12-216**

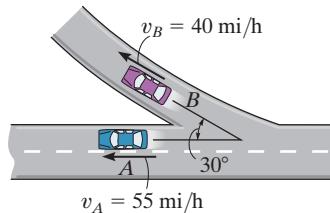
- 12-217.** The crate  $C$  is being lifted by moving the roller at  $A$  downward with a constant speed of  $v_A = 2 \text{ m/s}$  along the guide. Determine the velocity and acceleration of the crate at the instant  $s = 1 \text{ m}$ . When the roller is at  $B$ , the crate rests on the ground. Neglect the size of the pulley in the calculation. Hint: Relate the coordinates  $x_C$  and  $x_A$  using the problem geometry, then take the first and second time derivatives.

**Prob. 12-217**

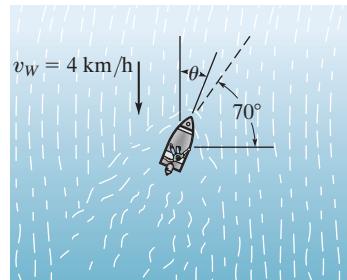
- 12-218.** Two planes,  $A$  and  $B$ , are flying at the same altitude. If their velocities are  $v_A = 500 \text{ km/h}$  and  $v_B = 700 \text{ km/h}$  such that the angle between their straight-line courses is  $\theta = 60^\circ$ , determine the velocity of plane  $B$  with respect to plane  $A$ .

**Prob. 12-218**

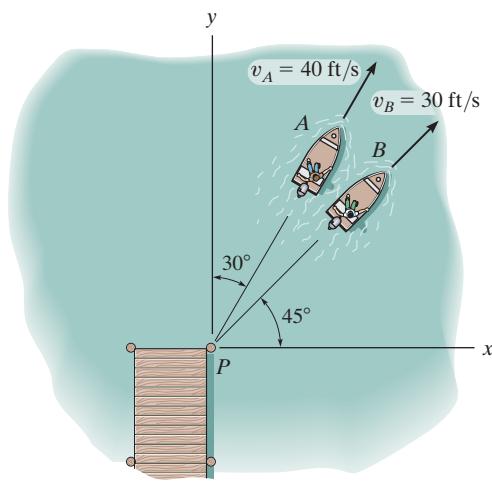
- 12-219.** At the instant shown, cars  $A$  and  $B$  are traveling at speeds of  $55 \text{ mi/h}$  and  $40 \text{ mi/h}$ , respectively. If  $B$  is increasing its speed by  $1200 \text{ mi/h}^2$ , while  $A$  maintains a constant speed, determine the velocity and acceleration of  $B$  with respect to  $A$ . Car  $B$  moves along a curve having a radius of curvature of  $0.5 \text{ mi}$ .

**Prob. 12-219**

- \***12-220.** The boat can travel with a speed of  $16 \text{ km/h}$  in still water. The point of destination is located along the dashed line. If the water is moving at  $4 \text{ km/h}$ , determine the bearing angle  $\theta$  at which the boat must travel to stay on course.

**Prob. 12-220**

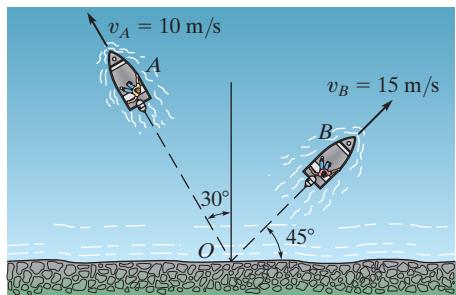
- 12–221.** Two boats leave the pier  $P$  at the same time and travel in the directions shown. If  $v_A = 40 \text{ ft/s}$  and  $v_B = 30 \text{ ft/s}$ , determine the velocity of boat  $A$  relative to boat  $B$ . How long after leaving the pier will the boats be 1500 ft apart?



Prob. 12–221

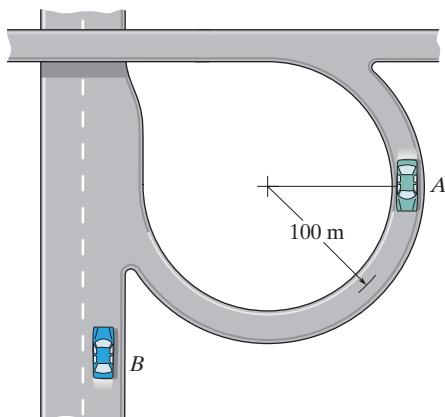
- 12–222.** A car is traveling north along a straight road at 50 km/h. An instrument in the car indicates that the wind is coming from the east. If the car's speed is 80 km/h, the instrument indicates that the wind is coming from the northeast. Determine the speed and direction of the wind.

- 12–223.** Two boats leave the shore at the same time and travel in the directions shown. If  $v_A = 10 \text{ m/s}$  and  $v_B = 15 \text{ m/s}$ , determine the velocity of boat  $A$  with respect to boat  $B$ . How long after leaving the shore will the boats be 600 m apart?



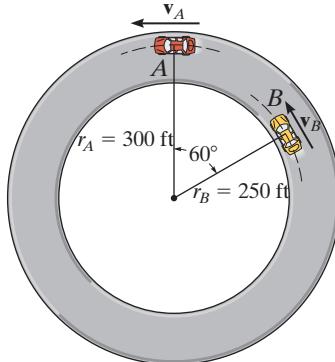
Prob. 12–223

- \*12–224.** At the instant shown, car  $A$  has a speed of 20 km/h, which is being increased at the rate of  $300 \text{ km/h}^2$  as the car enters the expressway. At the same instant, car  $B$  is decelerating at  $250 \text{ km/h}^2$  while traveling forward at 100 km/h. Determine the velocity and acceleration of  $A$  with respect to  $B$ .



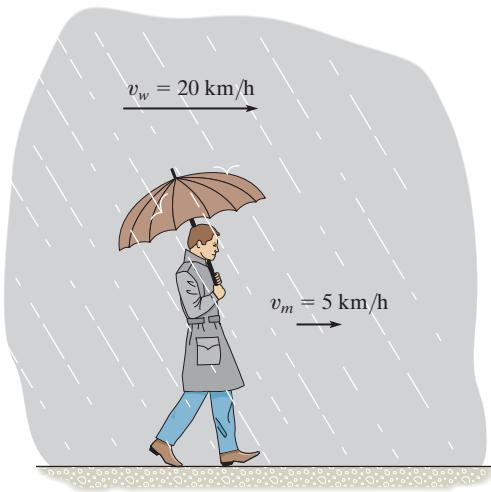
Prob. 12–224

- 12–225.** Cars  $A$  and  $B$  are traveling around the circular race track. At the instant shown,  $A$  has a speed of 90 ft/s and is increasing its speed at the rate of  $15 \text{ ft/s}^2$ , whereas  $B$  has a speed of 105 ft/s and is decreasing its speed at  $25 \text{ ft/s}^2$ . Determine the relative velocity and relative acceleration of car  $A$  with respect to car  $B$  at this instant.



Prob. 12–225

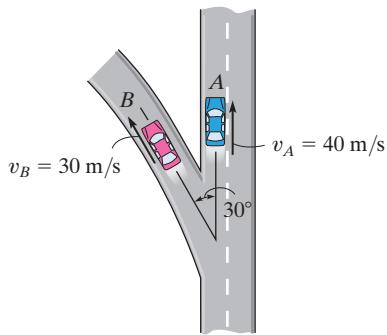
- 12-226.** A man walks at 5 km/h in the direction of a 20 km/h wind. If raindrops fall vertically at 7 km/h in *still air*, determine direction in which the drops appear to fall with respect to the man.



Prob. 12-226

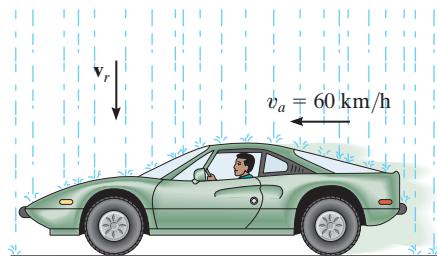
- 12-227.** At the instant shown, cars *A* and *B* are traveling at velocities of 40 m/s and 30 m/s, respectively. If *B* is increasing its velocity by  $2 \text{ m/s}^2$ , while *A* maintains a constant velocity, determine the velocity and acceleration of *B* with respect to *A*. The radius of curvature at *B* is  $\rho_B = 200 \text{ m}$ .

- \***12-228.** At the instant shown, cars *A* and *B* are traveling at velocities of 40 m/s and 30 m/s, respectively. If *A* is increasing its speed at  $4 \text{ m/s}^2$ , whereas the speed of *B* is decreasing at  $3 \text{ m/s}^2$ , determine the velocity and acceleration of *B* with respect to *A*. The radius of curvature at *B* is  $\rho_B = 200 \text{ m}$ .



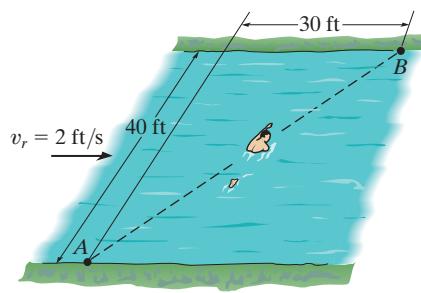
Probs. 12-227/228

- 12-229.** A passenger in an automobile observes that raindrops make an angle of  $30^\circ$  with the horizontal as the auto travels forward with a speed of 60 km/h. Compute the terminal (constant) velocity  $v_r$  of the rain if it is assumed to fall vertically.



Prob. 12-229

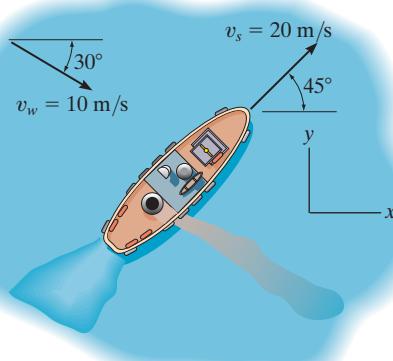
- 12-230.** A man can swim at 4 ft/s in still water. He wishes to cross the 40-ft-wide river to point *B*, 30 ft downstream. If the river flows with a velocity of 2 ft/s, determine the speed of the man and the time needed to make the crossing. Note: While in the water he must not direct himself toward point *B* to reach this point. Why?



Prob. 12-230

12

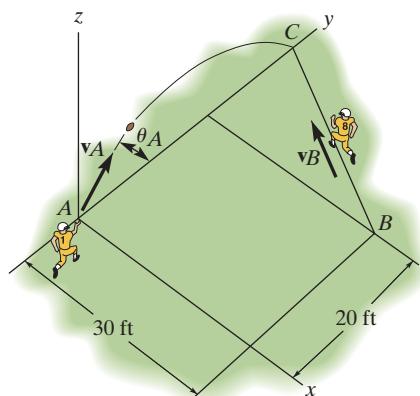
- 12-231.** The ship travels at a constant speed of  $v_s = 20 \text{ m/s}$  and the wind is blowing at a speed of  $v_w = 10 \text{ m/s}$ , as shown. Determine the magnitude and direction of the horizontal component of velocity of the smoke coming from the smoke stack as it appears to a passenger on the ship.



Prob. 12-231

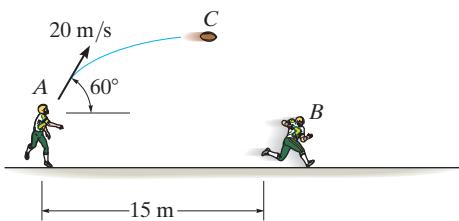
- \*12-232.** The football player at  $A$  throws the ball in the  $y-z$  plane at a speed  $v_A = 50 \text{ ft/s}$  and an angle  $\theta_A = 60^\circ$  with the horizontal. At the instant the ball is thrown, the player is at  $B$  and is running with constant speed along the line  $BC$  in order to catch it. Determine this speed,  $v_B$ , so that he makes the catch at the same elevation from which the ball was thrown.

- 12-233.** The football player at  $A$  throws the ball in the  $y-z$  plane with a speed  $v_A = 50 \text{ ft/s}$  and an angle  $\theta_A = 60^\circ$  with the horizontal. At the instant the ball is thrown, the player is at  $B$  and is running at a constant speed of  $v_B = 23 \text{ ft/s}$  along the line  $BC$ . Determine if he can reach point  $C$ , which has the same elevation as  $A$ , before the ball gets there.



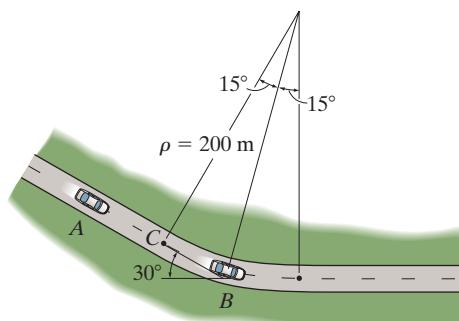
Probs. 12-232/233

- 12-234.** At a given instant the football player at  $A$  throws a football  $C$  with a velocity of  $20 \text{ m/s}$  in the direction shown. Determine the constant speed at which the player at  $B$  must run so that he can catch the football at the same elevation at which it was thrown. Also calculate the relative velocity and relative acceleration of the football with respect to  $B$  at the instant the catch is made. Player  $B$  is  $15 \text{ m}$  away from  $A$  when  $A$  starts to throw the football.



Prob. 12-234

- 12-235.** At the instant shown, car  $A$  travels along the straight portion of the road with a speed of  $25 \text{ m/s}$ . At this same instant car  $B$  travels along the circular portion of the road with a speed of  $15 \text{ m/s}$ . Determine the velocity of car  $B$  relative to car  $A$ .



Prob. 12-235

## CONCEPTUAL PROBLEMS

**C12-1.** If you measured the time it takes for the construction elevator to go from *A* to *B*, then *B* to *C*, and then *C* to *D*, and you also know the distance between each of the points, how could you determine the average velocity and average acceleration of the elevator as it ascends from *A* to *D*? Use numerical values to explain how this can be done.



**Prob. C12-1** (© R.C. Hibbeler)

**C12-2.** If the sprinkler at *A* is 1 m from the ground, then scale the necessary measurements from the photo to determine the approximate velocity of the water jet as it flows from the nozzle of the sprinkler.



**Prob. C12-2** (© R.C. Hibbeler)

**C12-3.** The basketball was thrown at an angle measured from the horizontal to the man's outstretched arm. If the basket is 3 m from the ground, make appropriate measurements in the photo and determine if the ball located as shown will pass through the basket.



**Prob. C12-3** (© R.C. Hibbeler)

**C12-4.** The pilot tells you the wingspan of her plane and her constant airspeed. How would you determine the acceleration of the plane at the moment shown? Use numerical values and take any necessary measurements from the photo.



**Prob. C12-4** (© R.C. Hibbeler)

## CHAPTER REVIEW

### Rectilinear Kinematics

Rectilinear kinematics refers to motion along a straight line. A position coordinate  $s$  specifies the location of the particle on the line, and the displacement  $\Delta s$  is the change in this position.

The average velocity is a vector quantity, defined as the displacement divided by the time interval.

The average speed is a scalar, and is the total distance traveled divided by the time of travel.

The time, position, velocity, and acceleration are related by three differential equations.

If the acceleration is known to be constant, then the differential equations relating time, position, velocity, and acceleration can be integrated.

### Graphical Solutions

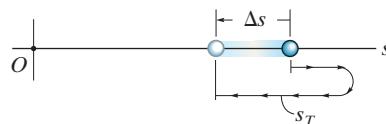
If the motion is erratic, then it can be described by a graph. If one of these graphs is given, then the others can be established using the differential relations between  $a$ ,  $v$ ,  $s$ , and  $t$ .

$$v_{\text{avg}} = \frac{\Delta s}{\Delta t}$$

$$(v_{\text{sp}})_{\text{avg}} = \frac{s_T}{\Delta t}$$

$$a = \frac{dv}{dt}, \quad v = \frac{ds}{dt}, \quad a ds = v dv$$

$$\begin{aligned} v &= v_0 + a_c t \\ s &= s_0 + v_0 t + \frac{1}{2} a_c t^2 \\ v^2 &= v_0^2 + 2a_c(s - s_0) \end{aligned}$$



$$\begin{aligned} a &= \frac{dv}{dt}, \\ v &= \frac{ds}{dt}, \\ a ds &= v dv \end{aligned}$$

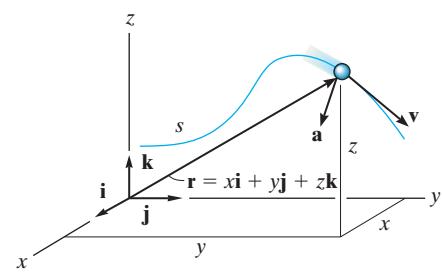
### Curvilinear Motion, $x, y, z$

Curvilinear motion along the path can be resolved into rectilinear motion along the  $x$ ,  $y$ ,  $z$  axes. The equation of the path is used to relate the motion along each axis.

$$v_x = \dot{x} \quad a_x = \dot{v}_x$$

$$v_y = \dot{y} \quad a_y = \dot{v}_y$$

$$v_z = \dot{z} \quad a_z = \dot{v}_z$$



### Projectile Motion

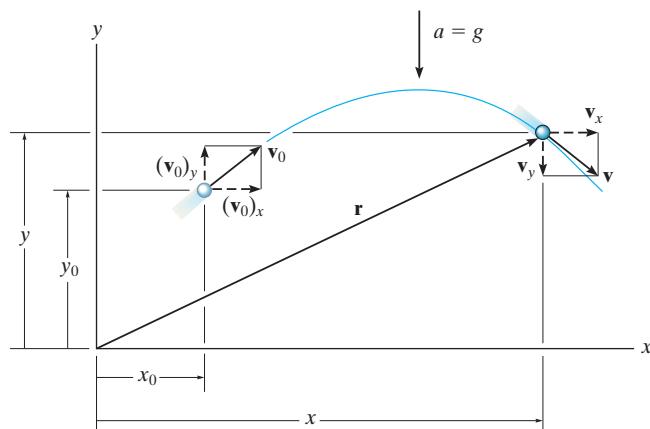
Free-flight motion of a projectile follows a parabolic path. It has a constant velocity in the horizontal direction, and a constant downward acceleration of  $g = 9.81 \text{ m/s}^2$  or  $32.2 \text{ ft/s}^2$  in the vertical direction. Any two of the three equations for constant acceleration apply in the vertical direction, and in the horizontal direction only one equation applies.

$$(+\uparrow) \quad v_y = (v_0)_y + a_c t$$

$$(+\uparrow) \quad y = y_0 + (v_0)_y t + \frac{1}{2} a_c t^2$$

$$(+\uparrow) \quad v_y^2 = (v_0)_y^2 + 2a_c(y - y_0)$$

$$(\pm) \quad x = x_0 + (v_0)_x t$$



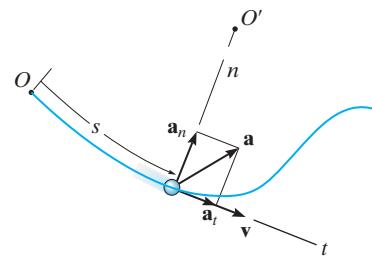
### Curvilinear Motion $n, t$

If normal and tangential axes are used for the analysis, then  $\mathbf{v}$  is always in the positive  $t$  direction.

The acceleration has two components. The tangential component,  $\mathbf{a}_t$ , accounts for the change in the magnitude of the velocity; a slowing down is in the negative  $t$  direction, and a speeding up is in the positive  $t$  direction. The normal component  $\mathbf{a}_n$  accounts for the change in the direction of the velocity. This component is always in the positive  $n$  direction.

$$a_t = \dot{v} \quad \text{or} \quad a_t ds = v dv$$

$$a_n = \frac{v^2}{\rho}$$



### Curvilinear Motion $r, \theta$

If the path of motion is expressed in polar coordinates, then the velocity and acceleration components can be related to the time derivatives of  $r$  and  $\theta$ .

To apply the time-derivative equations, it is necessary to determine  $r, \dot{r}, \ddot{r}, \theta, \dot{\theta}, \ddot{\theta}$  at the instant considered. If the path  $r = f(\theta)$  is given, then the chain rule of calculus must be used to obtain time derivatives. (See Appendix C.)

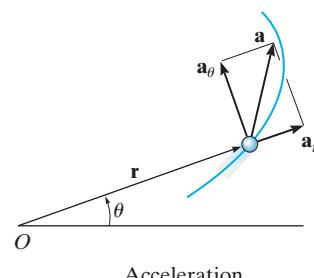
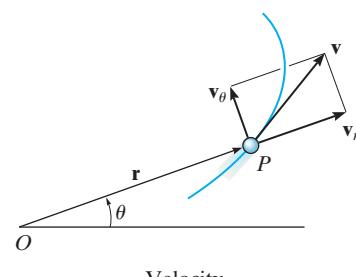
Once the data are substituted into the equations, then the algebraic sign of the results will indicate the direction of the components of  $\mathbf{v}$  or  $\mathbf{a}$  along each axis.

$$v_r = \dot{r}$$

$$v_\theta = r\dot{\theta}$$

$$a_r = \ddot{r} - r\dot{\theta}^2$$

$$a_\theta = r\ddot{\theta} + 2\dot{r}\dot{\theta}$$



### Absolute Dependent Motion of Two Particles

The dependent motion of blocks that are suspended from pulleys and cables can be related by the geometry of the system. This is done by first establishing position coordinates, measured from a fixed origin to each block. Each coordinate must be directed along the line of motion of a block.

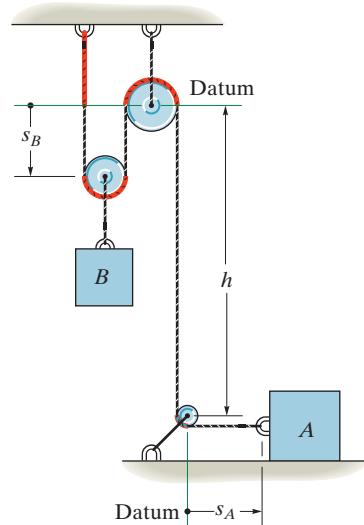
Using geometry and/or trigonometry, the coordinates are then related to the cable length in order to formulate a position coordinate equation.

The first time derivative of this equation gives a relationship between the velocities of the blocks, and a second time derivative gives the relation between their accelerations.

$$2s_B + h + s_A = l$$

$$2v_B = -v_A$$

$$2a_B = -a_A$$



### Relative-Motion Analysis Using Translating Axes

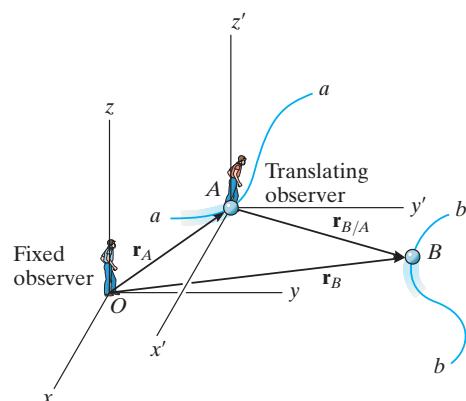
If two particles  $A$  and  $B$  undergo independent motions, then these motions can be related to their relative motion using a *translating set of axes* attached to one of the particles ( $A$ ).

For planar motion, each vector equation produces two scalar equations, one in the  $x$ , and the other in the  $y$  direction. For solution, the vectors can be expressed in Cartesian form, or the  $x$  and  $y$  scalar components can be written directly.

$$\mathbf{r}_B = \mathbf{r}_A + \mathbf{r}_{B/A}$$

$$\mathbf{v}_B = \mathbf{v}_A + \mathbf{v}_{B/A}$$

$$\mathbf{a}_B = \mathbf{a}_A + \mathbf{a}_{B/A}$$



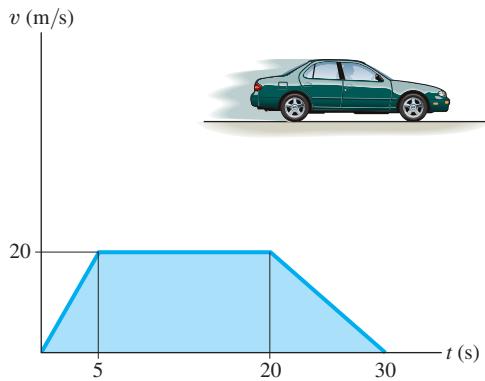
## REVIEW PROBLEMS

**R12–1.** The position of a particle along a straight line is given by  $s = (t^3 - 9t^2 + 15t)$  ft, where  $t$  is in seconds. Determine its maximum acceleration and maximum velocity during the time interval  $0 \leq t \leq 10$  s.

**R12–2.** If a particle has an initial velocity  $v_0 = 12$  ft/s to the right, and a constant acceleration of  $2$  ft/s $^2$  to the left, determine the particle's displacement in  $10$  s. Originally  $s_0 = 0$ .

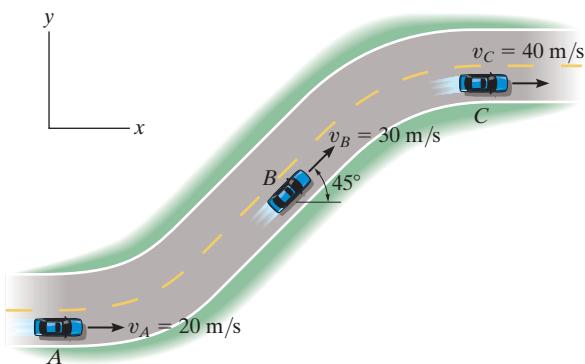
**R12–3.** A projectile, initially at the origin, moves along a straight-line path through a fluid medium such that its velocity is  $v = 1800(1 - e^{-0.3t})$  mm/s where  $t$  is in seconds. Determine the displacement of the projectile during the first  $3$  s.

**R12–4.** The  $v$ - $t$  graph of a car while traveling along a road is shown. Determine the acceleration when  $t = 2.5$  s,  $10$  s, and  $25$  s. Also if  $s = 0$  when  $t = 0$ , find the position when  $t = 5$  s,  $20$  s, and  $30$  s.



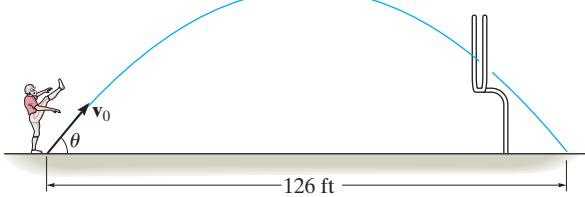
Prob. R12–4

**R12–5.** A car traveling along the straight portions of the road has the velocities indicated in the figure when it arrives at points  $A$ ,  $B$ , and  $C$ . If it takes  $3$  s to go from  $A$  to  $B$ , and then  $5$  s to go from  $B$  to  $C$ , determine the average acceleration between points  $A$  and  $B$  and between points  $A$  and  $C$ .



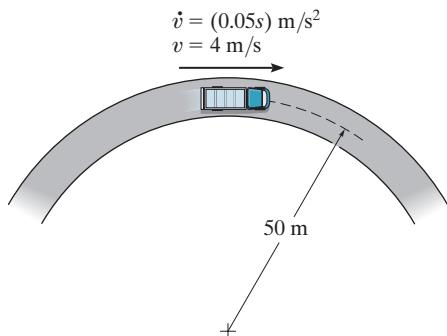
Prob. R12–5

**R12–6.** From a videotape, it was observed that a player kicked a football  $126$  ft during a measured time of  $3.6$  seconds. Determine the initial speed of the ball and the angle  $\theta$  at which it was kicked.



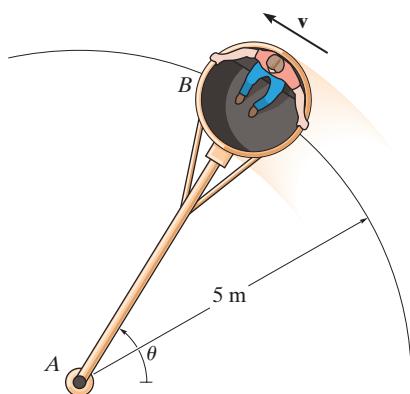
Prob. R12–6

**R12-7.** The truck travels in a circular path having a radius of 50 m at a speed of  $v = 4 \text{ m/s}$ . For a short distance from  $s = 0$ , its speed is increased by  $\dot{v} = (0.05s) \text{ m/s}^2$ , where  $s$  is in meters. Determine its speed and the magnitude of its acceleration when it has moved  $s = 10 \text{ m}$ .



Prob. R12-7

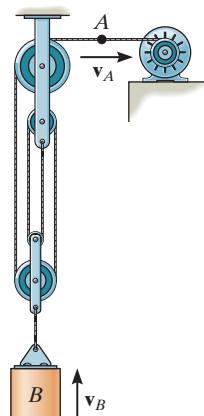
**R12-8.** Car  $B$  turns such that its speed is increased by  $(a_t)_B = (0.5e^t) \text{ m/s}^2$ , where  $t$  is in seconds. If the car starts from rest when  $\theta = 0^\circ$ , determine the magnitudes of its velocity and acceleration when  $t = 2 \text{ s}$ . Neglect the size of the car.



Prob. R12-8

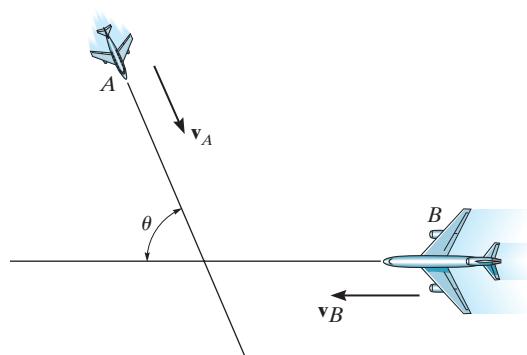
**R12-9.** A particle is moving along a circular path of 2-m radius such that its position as a function of time is given by  $\theta = (5t^2) \text{ rad}$ , where  $t$  is in seconds. Determine the magnitude of the particle's acceleration when  $\theta = 30^\circ$ . The particle starts from rest when  $\theta = 0^\circ$ .

**R12-10.** Determine the time needed for the load at  $B$  to attain a speed of 8 m/s, starting from rest, if the cable is drawn into the motor with an acceleration of  $0.2 \text{ m/s}^2$ .



Prob. R12-10

**R12-11.** Two planes,  $A$  and  $B$ , are flying at the same altitude. If their velocities are  $v_A = 600 \text{ km/h}$  and  $v_B = 500 \text{ km/h}$  such that the angle between their straight-line courses is  $\theta = 75^\circ$ , determine the velocity of plane  $B$  with respect to plane  $A$ .



Prob. R12-11

# Chapter 13



(© Migel/Shutterstock)

A car driving along this road will be subjected to forces that create both normal and tangential accelerations. In this chapter we will study how these forces are related to the accelerations they create.

# Kinetics of a Particle: Force and Acceleration

## CHAPTER OBJECTIVES

- To state Newton's Second Law of Motion and to define mass and weight.
- To analyze the accelerated motion of a particle using the equation of motion with different coordinate systems.
- To investigate central-force motion and apply it to problems in space mechanics.

### 13.1 Newton's Second Law of Motion

*Kinetics* is a branch of dynamics that deals with the relationship between the change in motion of a body and the forces that cause this change. The basis for kinetics is Newton's second law, which states that when an *unbalanced force* acts on a particle, the particle will *accelerate* in the direction of the force with a magnitude that is proportional to the force.

This law can be verified experimentally by applying a known unbalanced force  $\mathbf{F}$  to a particle, and then measuring the acceleration  $\mathbf{a}$ . Since the force and acceleration are directly proportional, the constant of proportionality,  $m$ , may be determined from the ratio  $m = F/a$ . This positive scalar  $m$  is called the *mass* of the particle. Being constant during any acceleration,  $m$  provides a quantitative measure of the resistance of the particle to a change in its velocity, that is its *inertia*.



The jeep leans backward due to its inertia, which resists its forward acceleration.  
(© R.C. Hibbeler)

If the mass of the particle is  $m$ , Newton's second law of motion may be written in mathematical form as

$$\mathbf{F} = m\mathbf{a}$$

The above equation, which is referred to as the *equation of motion*, is one of the most important formulations in mechanics.\* As previously stated, its validity is based solely on *experimental evidence*. In 1905, however, Albert Einstein developed the theory of relativity and placed limitations on the use of Newton's second law for describing general particle motion. Through experiments it was proven that *time* is not an absolute quantity as assumed by Newton; and as a result, the equation of motion fails to predict the exact behavior of a particle, especially when the particle's speed approaches the speed of light (0.3 Gm/s). Developments of the theory of quantum mechanics by Erwin Schrödinger and others indicate further that conclusions drawn from using this equation are also invalid when particles are the size of an atom and move close to one another. For the most part, however, these requirements regarding particle speed and size are not encountered in engineering problems, so their effects will not be considered in this book.

**Newton's Law of Gravitational Attraction.** Shortly after formulating his three laws of motion, Newton postulated a law governing the mutual attraction between any two particles. In mathematical form this law can be expressed as

$$F = G \frac{m_1 m_2}{r^2} \quad (13-1)$$

where

- $F$  = force of attraction between the two particles
- $G$  = universal constant of gravitation; according to experimental evidence  $G = 66.73(10^{-12}) \text{ m}^3/(\text{kg} \cdot \text{s}^2)$
- $m_1, m_2$  = mass of each of the two particles
- $r$  = distance between the centers of the two particles

\*Since  $m$  is constant, we can also write  $\mathbf{F} = d(m\mathbf{v})/dt$ , where  $m\mathbf{v}$  is the particle's linear momentum. Here the unbalanced force acting on the particle is proportional to the time rate of change of the particle's linear momentum.

In the case of a particle located at or near the surface of the earth, the only gravitational force having any sizable magnitude is that between the earth and the particle. This force is termed the “weight” and, for our purpose, it will be the only gravitational force considered.

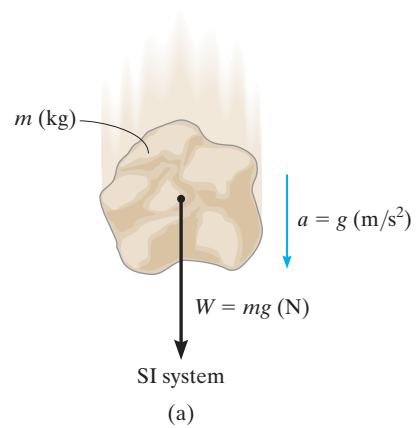
From Eq. 13-1, we can develop a general expression for finding the weight  $W$  of a particle having a mass  $m_1 = m$ . Let  $m_2 = M_e$  be the mass of the earth and  $r$  the distance between the earth's center and the particle. Then, if  $g = GM_e/r^2$ , we have

$$W = mg$$

By comparison with  $F = ma$ , we term  $g$  the acceleration due to gravity. For most engineering calculations  $g$  is measured at a point on the surface of the earth at sea level, and at a latitude of  $45^\circ$ , which is considered the “standard location.” Here the values  $g = 9.81 \text{ m/s}^2 = 32.2 \text{ ft/s}^2$  will be used for calculations.

In the SI system the mass of the body is specified in kilograms, and the weight must be calculated using the above equation, Fig. 13-1a. Thus,

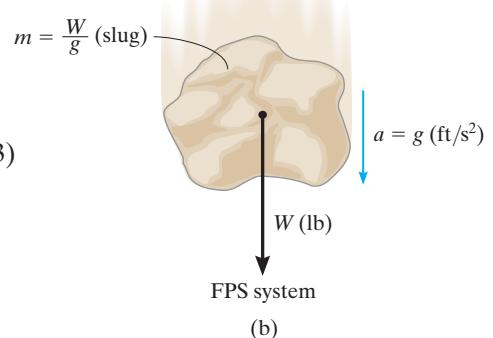
$$W = mg \text{ (N)} \quad (g = 9.81 \text{ m/s}^2) \quad (13-2)$$



As a result, a body of mass 1 kg has a weight of 9.81 N; a 2-kg body weighs 19.62 N; and so on.

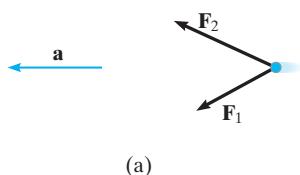
In the FPS system the weight of the body is specified in pounds. The mass is measured in slugs, a term derived from “sluggish” which refers to the body's inertia. It must be calculated, Fig. 13-1b, using

$$m = \frac{W}{g} \text{ (slug)} \quad (g = 32.2 \text{ ft/s}^2) \quad (13-3)$$



Therefore, a body weighing 32.2 lb has a mass of 1 slug; a 64.4-lb body has a mass of 2 slugs; and so on.

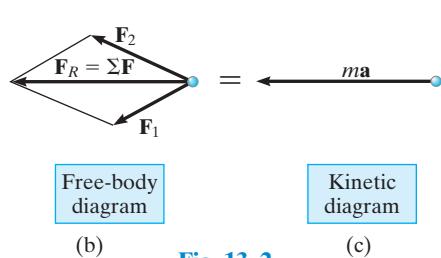
**Fig. 13-1**



## 13.2 The Equation of Motion

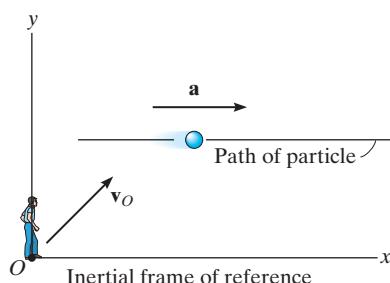
When more than one force acts on a particle, the resultant force is determined by a vector summation of all the forces; i.e.,  $\mathbf{F}_R = \Sigma \mathbf{F}$ . For this more general case, the equation of motion may be written as

$$\sum \mathbf{F} = ma \quad (13-4)$$



**Fig. 13-2**

To illustrate application of this equation, consider the particle shown in Fig. 13-2a, which has a mass  $m$  and is subjected to the action of two forces,  $\mathbf{F}_1$  and  $\mathbf{F}_2$ . We can graphically account for the magnitude and direction of each force acting on the particle by drawing the particle's *free-body diagram*, Fig. 13-2b. Since the *resultant* of these forces produces the vector  $m\mathbf{a}$ , its magnitude and direction can be represented graphically on the *kinetic diagram*, shown in Fig. 13-2c.\* The equal sign written between the diagrams symbolizes the *graphical* equivalency between the free-body diagram and the kinetic diagram; i.e.,  $\Sigma \mathbf{F} = m\mathbf{a}$ .† In particular, note that if  $\mathbf{F}_R = \Sigma \mathbf{F} = \mathbf{0}$ , then the acceleration is also zero, so that the particle will either remain at *rest* or move along a straight-line path with *constant velocity*. Such are the conditions of *static equilibrium*, Newton's first law of motion.



**Fig. 13–3**

**Inertial Reference Frame.** When applying the equation of motion, it is important that the acceleration of the particle be measured with respect to a reference frame that is *either fixed or translates with a constant velocity*. In this way, the observer will not accelerate and measurements of the particle's acceleration will be the *same* from *any* reference of this type. Such a frame of reference is commonly known as a *Newtonian* or *inertial reference frame*. Fig. 13-3.

When studying the motions of rockets and satellites, it is justifiable to consider the inertial reference frame as fixed to the stars, whereas dynamics problems concerned with motions on or near the surface of the earth may be solved by using an inertial frame which is assumed fixed to the earth. Even though the earth both rotates about its own axis and revolves about the sun, the accelerations created by these rotations are relatively small and so they can be neglected for most applications.

\*Recall the free-body diagram considers the particle to be free of its surrounding supports and shows all the forces acting on the particle. The kinetic diagram pertains to the particle's motion as caused by the forces.

<sup>†</sup>The equation of motion can also be rewritten in the form  $\Sigma \mathbf{F} - m\mathbf{a} = \mathbf{0}$ . The vector  $-m\mathbf{a}$  is referred to as the *inertia force vector*. If it is treated in the same way as a “force vector,” then the state of “equilibrium” created is referred to as *dynamic equilibrium*. This method of application, which will not be used in this text, is often referred to as the *D'Alembert principle*, named after the French mathematician Jean le Rond d'Alembert.

We are all familiar with the sensation one feels when sitting in a car that is subjected to a forward acceleration. Often people think this is caused by a “force” which acts on them and tends to push them back in their seats; however, this is not the case. Instead, this sensation occurs due to their inertia or the resistance of their mass to a change in velocity.

Consider the passenger who is strapped to the seat of a rocket sled. Provided the sled is at rest or is moving with constant velocity, then no force is exerted on his back as shown on his free-body diagram.



Keystone/Hulton Archive/  
Getty Images

When the thrust of the rocket engine causes the sled to accelerate, then the seat upon which he is sitting exerts a force  $\mathbf{F}$  on him which pushes him forward with the sled. In the photo, notice that the inertia of his head resists this change in motion (acceleration), and so his head moves back against the seat and his face, which is nonrigid, tends to distort backward.



Keystone/Hulton Archive/  
Getty Images

Upon deceleration the force of the seatbelt  $\mathbf{F}'$  tends to pull his body to a stop, but his head leaves contact with the back of the seat and his face distorts forward, again due to his inertia or tendency to continue to move forward. No force is pulling him forward, although this is the sensation he receives.



Keystone/Hulton Archive/  
Getty Images

### 13.3 Equation of Motion for a System of Particles

The equation of motion will now be extended to include a system of particles isolated within an enclosed region in space, as shown in Fig. 13–4a. In particular, there is no restriction in the way the particles are connected, so the following analysis applies equally well to the motion of a solid, liquid, or gas system.

At the instant considered, the arbitrary  $i$ -th particle, having a mass  $m_i$ , is subjected to a system of internal forces and a resultant external force. The *internal force*, represented symbolically as  $\mathbf{f}_i$ , is the resultant of all the forces the other particles exert on the  $i$ th particle. The *resultant external force*  $\mathbf{F}_i$  represents, for example, the effect of gravitational, electrical, magnetic, or contact forces between the  $i$ th particle and adjacent bodies or particles *not* included within the system.

The free-body and kinetic diagrams for the  $i$ th particle are shown in Fig. 13–4b. Applying the equation of motion,

$$\Sigma \mathbf{F} = m\mathbf{a}; \quad \mathbf{F}_i + \mathbf{f}_i = m_i \mathbf{a}_i$$

When the equation of motion is applied to each of the other particles of the system, similar equations will result. And, if all these equations are added together *vectorially*, we obtain

$$\Sigma \mathbf{F}_i + \Sigma \mathbf{f}_i = \Sigma m_i \mathbf{a}_i$$

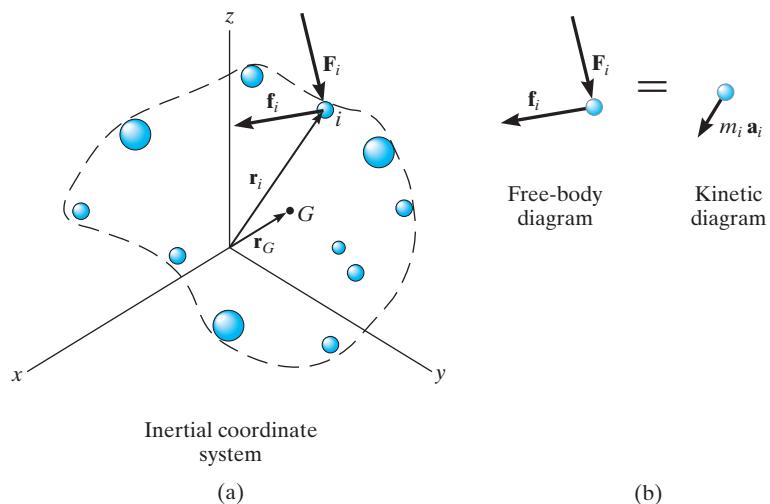


Fig. 13–4

The summation of the internal forces, if carried out, will equal zero, since internal forces between any two particles occur in equal but opposite collinear pairs. Consequently, only the sum of the external forces will remain, and therefore the equation of motion, written for the system of particles, becomes

$$\sum \mathbf{F}_i = \sum m_i \mathbf{a}_i \quad (13-5)$$

If  $\mathbf{r}_G$  is a position vector which locates the *center of mass*  $G$  of the particles, Fig. 13–4a, then by definition of the center of mass,  $m\mathbf{r}_G = \sum m_i \mathbf{r}_i$ , where  $m = \sum m_i$  is the total mass of all the particles. Differentiating this equation twice with respect to time, assuming that no mass is entering or leaving the system, yields

$$m\mathbf{a}_G = \sum m_i \mathbf{a}_i$$

Substituting this result into Eq. 13–5, we obtain

$$\sum \mathbf{F} = m\mathbf{a}_G \quad (13-6)$$

Hence, the sum of the external forces acting on the system of particles is equal to the total mass of the particles times the acceleration of its center of mass  $G$ . Since in reality all particles must have a finite size to possess mass, Eq. 13–6 justifies application of the equation of motion to a *body* that is represented as a single particle.

### Important Points

- The equation of motion is based on experimental evidence and is valid only when applied within an inertial frame of reference.
- The equation of motion states that the *unbalanced force* on a particle causes it to *accelerate*.
- An inertial frame of reference does not rotate, rather its axes either translate with constant velocity or are at rest.
- Mass is a property of matter that provides a quantitative measure of its resistance to a change in velocity. It is an absolute quantity and so it does not change from one location to another.
- Weight is a force that is caused by the earth's gravitation. It is not absolute; rather it depends on the altitude of the mass from the earth's surface.

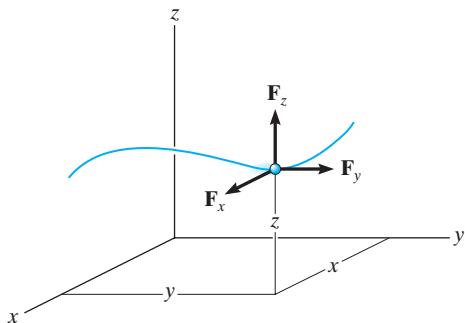
## 13.4 Equations of Motion: Rectangular Coordinates

When a particle moves relative to an inertial  $x, y, z$  frame of reference, the forces acting on the particle, as well as its acceleration, can be expressed in terms of their  $\mathbf{i}, \mathbf{j}, \mathbf{k}$  components, Fig. 13–5. Applying the equation of motion, we have

$$\Sigma \mathbf{F} = m\mathbf{a}; \quad \Sigma F_x \mathbf{i} + \Sigma F_y \mathbf{j} + \Sigma F_z \mathbf{k} = m(a_x \mathbf{i} + a_y \mathbf{j} + a_z \mathbf{k})$$

For this equation to be satisfied, the respective  $\mathbf{i}, \mathbf{j}, \mathbf{k}$  components on the left side must equal the corresponding components on the right side. Consequently, we may write the following three scalar equations:

$$\begin{aligned}\Sigma F_x &= ma_x \\ \Sigma F_y &= ma_y \\ \Sigma F_z &= ma_z\end{aligned}\tag{13-7}$$



**Fig. 13-5**

In particular, if the particle is constrained to move only in the  $x-y$  plane, then the first two of these equations are used to specify the motion.

### Procedure for Analysis

The equations of motion are used to solve problems which require a relationship between the forces acting on a particle and the accelerated motion they cause.

#### Free-Body Diagram.

- Select the inertial coordinate system. Most often, rectangular or  $x, y, z$  coordinates are chosen to analyze problems for which the particle has *rectilinear motion*.
- Once the coordinates are established, draw the particle's free-body diagram. Drawing this diagram is *very important* since it provides a graphical representation that accounts for *all the forces* ( $\Sigma \mathbf{F}$ ) which act on the particle, and thereby makes it possible to resolve these forces into their  $x, y, z$  components.
- The direction and sense of the particle's acceleration  $\mathbf{a}$  should also be established. If the sense is unknown, for mathematical convenience assume that the sense of each acceleration component acts in the *same direction* as its *positive* inertial coordinate axis.
- The acceleration may be represented as the  $m\mathbf{a}$  vector on the kinetic diagram.\*
- Identify the unknowns in the problem.

\*It is a convention in this text always to use the kinetic diagram as a graphical aid when developing the proofs and theory. The particle's acceleration or its components will be shown as blue colored vectors near the free-body diagram in the examples.

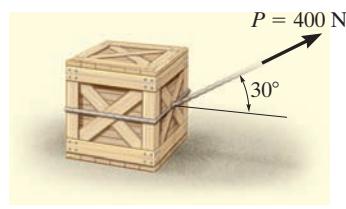
**Equations of Motion.**

- If the forces can be resolved directly from the free-body diagram, apply the equations of motion in their scalar component form.
- If the geometry of the problem appears complicated, which often occurs in three dimensions, Cartesian vector analysis can be used for the solution.
- *Friction.* If a moving particle contacts a rough surface, it may be necessary to use the *frictional equation*, which relates the frictional and normal forces  $\mathbf{F}_f$  and  $\mathbf{N}$  acting at the surface of contact by using the coefficient of kinetic friction, i.e.,  $F_f = \mu_k N$ . Remember that  $\mathbf{F}_f$  always acts on the free-body diagram such that it opposes the motion of the particle relative to the surface it contacts. If the particle is *on the verge* of relative motion, then the coefficient of static friction should be used.
- *Spring.* If the particle is connected to an *elastic spring* having negligible mass, the spring force  $F_s$  can be related to the deformation of the spring by the equation  $F_s = ks$ . Here  $k$  is the spring's stiffness measured as a force per unit length, and  $s$  is the stretch or compression defined as the difference between the deformed length  $l$  and the undeformed length  $l_0$ , i.e.,  $s = l - l_0$ .

**Kinematics.**

- If the velocity or position of the particle is to be found, it will be necessary to apply the necessary kinematic equations once the particle's acceleration is determined from  $\Sigma \mathbf{F} = m\mathbf{a}$ .
- If *acceleration* is a function of time, use  $a = dv/dt$  and  $v = ds/dt$  which, when integrated, yield the particle's velocity and position, respectively.
- If *acceleration* is a function of displacement, integrate  $a ds = v dv$  to obtain the velocity as a function of position.
- If *acceleration is constant*, use  $v = v_0 + a_c t$ ,  $s = s_0 + v_0 t + \frac{1}{2} a_c t^2$ ,  $v^2 = v_0^2 + 2a_c(s - s_0)$  to determine the velocity or position of the particle.
- If the problem involves the dependent motion of several particles, use the method outlined in Sec. 12.9 to relate their accelerations. In all cases, make sure the positive inertial coordinate directions used for writing the kinematic equations are the same as those used for writing the equations of motion; otherwise, simultaneous solution of the equations will result in errors.
- If the solution for an unknown vector component yields a negative scalar, it indicates that the component acts in the direction opposite to that which was assumed.

## EXAMPLE | 13.1



(a)

The 50-kg crate shown in Fig. 13–6a rests on a horizontal surface for which the coefficient of kinetic friction is  $\mu_k = 0.3$ . If the crate is subjected to a 400-N towing force as shown, determine the velocity of the crate in 3 s starting from rest.

**SOLUTION**

Using the equations of motion, we can relate the crate's acceleration to the force causing the motion. The crate's velocity can then be determined using kinematics.

**Free-Body Diagram.** The weight of the crate is  $W = mg = 50 \text{ kg} (9.81 \text{ m/s}^2) = 490.5 \text{ N}$ . As shown in Fig. 13–6b, the frictional force has a magnitude  $F = \mu_k N_C$  and acts to the left, since it opposes the motion of the crate. The acceleration  $\mathbf{a}$  is assumed to act horizontally, in the positive  $x$  direction. There are two unknowns, namely  $N_C$  and  $a$ .

**Equations of Motion.** Using the data shown on the free-body diagram, we have

$$\pm \sum F_x = ma_x; \quad 400 \cos 30^\circ - 0.3N_C = 50a \quad (1)$$

$$+\uparrow \sum F_y = ma_y; \quad N_C - 490.5 + 400 \sin 30^\circ = 0 \quad (2)$$

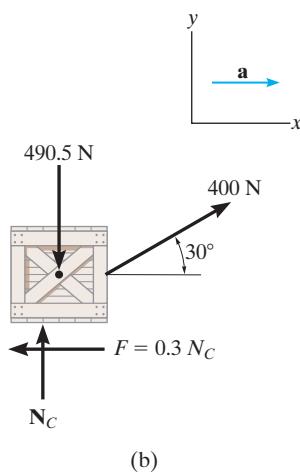
Solving Eq. 2 for  $N_C$ , substituting the result into Eq. 1, and solving for  $a$  yields

$$N_C = 290.5 \text{ N}$$

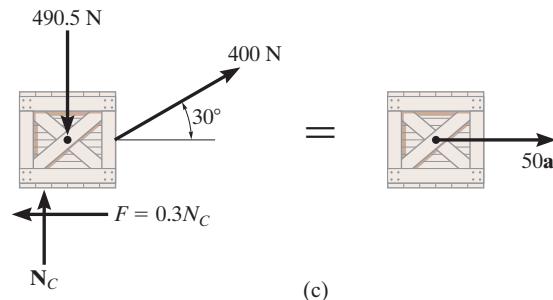
$$a = 5.185 \text{ m/s}^2$$

**Kinematics.** Notice that the acceleration is *constant*, since the applied force  $\mathbf{P}$  is constant. Since the initial velocity is zero, the velocity of the crate in 3 s is

$$( \Rightarrow ) \quad v = v_0 + a_c t = 0 + 5.185(3) \\ = 15.6 \text{ m/s} \rightarrow \quad \text{Ans.}$$



(b)

**Fig. 13–6**

**NOTE:** We can also use the alternative procedure of drawing the crate's free-body *and* kinetic diagrams, Fig. 13–6c, prior to applying the equations of motion.

**EXAMPLE | 13.2**

A 10-kg projectile is fired vertically upward from the ground, with an initial velocity of 50 m/s, Fig. 13–7a. Determine the maximum height to which it will travel if (a) atmospheric resistance is neglected; and (b) atmospheric resistance is measured as  $F_D = (0.01v^2)$  N, where  $v$  is the speed of the projectile at any instant, measured in m/s.

**SOLUTION**

In both cases the known force on the projectile can be related to its acceleration using the equation of motion. Kinematics can then be used to relate the projectile's acceleration to its position.

**Part (a) Free-Body Diagram.** As shown in Fig. 13–7b, the projectile's weight is  $W = mg = 10(9.81) = 98.1$  N. We will assume the unknown acceleration  $\mathbf{a}$  acts upward in the *positive z* direction.

**Equation of Motion.**

$$+\uparrow \sum F_z = ma_z; \quad -98.1 = 10a, \quad a = -9.81 \text{ m/s}^2$$

The result indicates that the projectile, like every object having free-flight motion near the earth's surface, is subjected to a *constant* downward acceleration of 9.81 m/s<sup>2</sup>.

**Kinematics.** Initially,  $z_0 = 0$  and  $v_0 = 50$  m/s, and at the maximum height  $z = h$ ,  $v = 0$ . Since the acceleration is *constant*, then

$$\begin{aligned} (+\uparrow) \quad v^2 &= v_0^2 + 2a_c(z - z_0) \\ &0 = (50)^2 + 2(-9.81)(h - 0) \\ h &= 127 \text{ m} \end{aligned} \quad \text{Ans.}$$

**Part (b) Free-Body Diagram.** Since the force  $F_D = (0.01v^2)$  N tends to retard the upward motion of the projectile, it acts downward as shown on the free-body diagram, Fig. 13–7c.

**Equation of Motion.**

$$+\uparrow \sum F_z = ma_z; \quad -0.01v^2 - 98.1 = 10a, \quad a = -(0.001v^2 + 9.81)$$

**Kinematics.** Here the acceleration is *not constant* since  $F_D$  depends on the velocity. Since  $a = f(v)$ , we can relate  $a$  to position using

$$(+\uparrow) a dz = v dv; \quad -(0.001v^2 + 9.81) dz = v dv$$

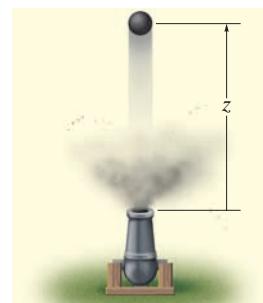
Separating the variables and integrating, realizing that initially  $z_0 = 0$ ,  $v_0 = 50$  m/s (positive upward), and at  $z = h$ ,  $v = 0$ , we have

$$\int_0^h dz = - \int_{50 \text{ m/s}}^0 \frac{v dv}{0.001v^2 + 9.81} = -500 \ln(v^2 + 9810) \Big|_{50 \text{ m/s}}^0$$

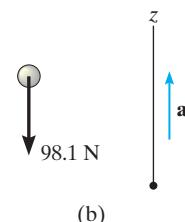
$$h = 114 \text{ m}$$

*Ans.*

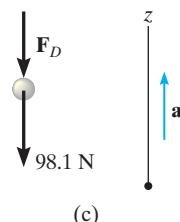
**NOTE:** The answer indicates a lower elevation than that obtained in part (a) due to atmospheric resistance or drag.



(a)



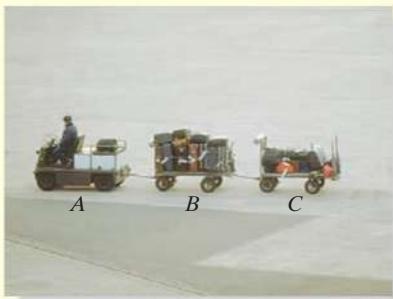
(b)



(c)

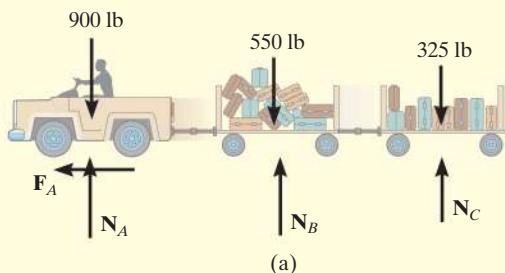
**Fig. 13–7**

## EXAMPLE | 13.3



(© R.C. Hibbeler)

The baggage truck *A* shown in the photo has a weight of 900 lb and tows a 550-lb cart *B* and a 325-lb cart *C*. For a short time the driving frictional force developed at the wheels of the truck is  $F_A = (40t)$  lb, where  $t$  is in seconds. If the truck starts from rest, determine its speed in 2 seconds. Also, what is the horizontal force acting on the coupling between the truck and cart *B* at this instant? Neglect the size of the truck and carts.



## SOLUTION

**Free-Body Diagram.** As shown in Fig. 13-8a, it is the frictional driving force that gives both the truck and carts an acceleration. Here we have considered all three vehicles as a single system.

**Equation of Motion.** Only motion in the horizontal direction has to be considered.

$$\pm \sum F_x = ma_x; \quad 40t = \left( \frac{900 + 550 + 325}{32.2} \right) a \\ a = 0.7256t$$

**Kinematics.** Since the acceleration is a function of time, the velocity of the truck is obtained using  $a = dv/dt$  with the initial condition that  $v_0 = 0$  at  $t = 0$ . We have

$$\int_0^v dv = \int_0^{2\text{ s}} 0.7256t dt; \quad v = 0.3628t^2 \Big|_0^{2\text{ s}} = 1.45 \text{ ft/s} \quad \text{Ans.}$$

**Free-Body Diagram.** In order to determine the force between the truck and cart *B*, we will consider a free-body diagram of the truck so that we can “expose” the coupling force  $\mathbf{T}$  as external to the free-body diagram, Fig. 13-8b.

**Equation of Motion.** When  $t = 2$  s, then

$$\pm \sum F_x = ma_x; \quad 40(2) - T = \left( \frac{900}{32.2} \right) [0.7256(2)] \\ T = 39.4 \text{ lb} \quad \text{Ans.}$$

**NOTE:** Try and obtain this same result by considering a free-body diagram of carts *B* and *C* as a single system.

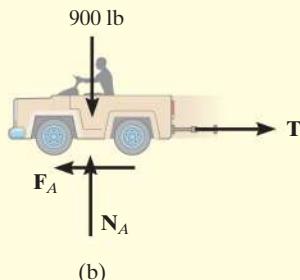


Fig. 13-8

**EXAMPLE | 13.4**

A smooth 2-kg collar, shown in Fig. 13–9a, is attached to a spring having a stiffness  $k = 3 \text{ N/m}$  and an unstretched length of 0.75 m. If the collar is released from rest at  $A$ , determine its acceleration and the normal force of the rod on the collar at the instant  $y = 1 \text{ m}$ .

**SOLUTION**

**Free-Body Diagram.** The free-body diagram of the collar when it is located at the arbitrary position  $y$  is shown in Fig. 13–9b. Furthermore, the collar is *assumed* to be accelerating so that “ $\mathbf{a}$ ” acts downward in the *positive*  $y$  direction. There are four unknowns, namely,  $N_C$ ,  $F_s$ ,  $a$ , and  $\theta$ .

**Equations of Motion.**

$$\pm \sum F_x = ma_x; \quad -N_C + F_s \cos \theta = 0 \quad (1)$$

$$+\downarrow \sum F_y = ma_y; \quad 19.62 - F_s \sin \theta = 2a \quad (2)$$

From Eq. 2 it is seen that the acceleration depends on the magnitude and direction of the spring force. Solution for  $N_C$  and  $a$  is possible once  $F_s$  and  $\theta$  are known.

The magnitude of the spring force is a function of the stretch  $s$  of the spring; i.e.,  $F_s = ks$ . Here the unstretched length is  $AB = 0.75 \text{ m}$ , Fig. 13–9a; therefore,  $s = CB - AB = \sqrt{y^2 + (0.75)^2} - 0.75$ . Since  $k = 3 \text{ N/m}$ , then

$$F_s = ks = 3 \left( \sqrt{y^2 + (0.75)^2} - 0.75 \right) \quad (3)$$

From Fig. 13–9a, the angle  $\theta$  is related to  $y$  by trigonometry.

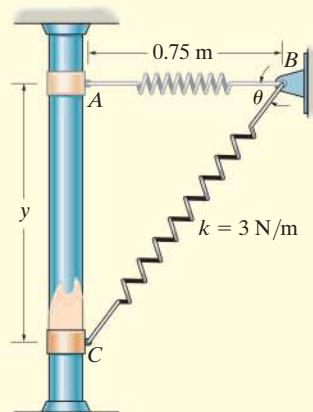
$$\tan \theta = \frac{y}{0.75}$$

Substituting  $y = 1 \text{ m}$  into Eqs. 3 and 4 yields  $F_s = 1.50 \text{ N}$  and  $\theta = 53.1^\circ$ . Substituting these results into Eqs. 1 and 2, we obtain

$$N_C = 0.900 \text{ N} \quad \text{Ans.}$$

$$a = 9.21 \text{ m/s}^2 \downarrow \quad \text{Ans.}$$

**NOTE:** This is not a case of constant acceleration, since the spring force changes both its magnitude and direction as the collar moves downward.



(a)

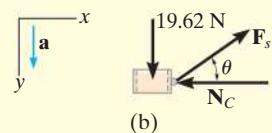
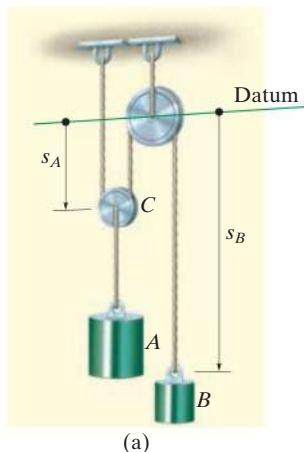


Fig. 13–9

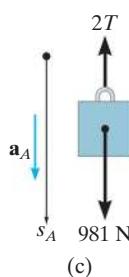
## EXAMPLE | 13.5



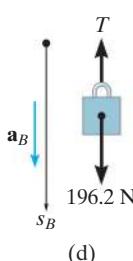
(a)



(b)



(c)



(d)

The 100-kg block *A* shown in Fig. 13–10*a* is released from rest. If the masses of the pulleys and the cord are neglected, determine the velocity of the 20-kg block *B* in 2 s.

## SOLUTION

**Free-Body Diagrams.** Since the mass of the pulleys is *neglected*, then for pulley *C*,  $ma = 0$  and we can apply  $\sum F_y = 0$ , as shown in Fig. 13–10*b*. The free-body diagrams for blocks *A* and *B* are shown in Fig. 13–10*c* and *d*, respectively. Notice that for *A* to remain stationary  $T = 490.5$  N, whereas for *B* to remain static  $T = 196.2$  N. Hence *A* will move down while *B* moves up. Although this is the case, we will assume both blocks accelerate downward, in the direction of  $+s_A$  and  $+s_B$ . The three unknowns are  $T$ ,  $a_A$ , and  $a_B$ .

**Equations of Motion.** Block *A*,

$$+\downarrow \sum F_y = ma_y; \quad 981 - 2T = 100a_A \quad (1)$$

Block *B*,

$$+\downarrow \sum F_y = ma_y; \quad 196.2 - T = 20a_B \quad (2)$$

**Kinematics.** The necessary third equation is obtained by relating  $a_A$  to  $a_B$  using a dependent motion analysis, discussed in Sec. 12.9. The coordinates  $s_A$  and  $s_B$  in Fig. 13–10*a* measure the positions of *A* and *B* from the fixed datum. It is seen that

$$2s_A + s_B = l$$

where *l* is constant and represents the total vertical length of cord. Differentiating this expression twice with respect to time yields

$$2a_A = -a_B \quad (3)$$

Notice that when writing Eqs. 1 to 3, the *positive direction was always assumed downward*. It is very important to be *consistent* in this assumption since we are seeking a simultaneous solution of equations. The results are

$$T = 327.0 \text{ N}$$

$$a_A = 3.27 \text{ m/s}^2$$

$$a_B = -6.54 \text{ m/s}^2$$

Hence when block *A* accelerates *downward*, block *B* accelerates *upward* as expected. Since  $a_B$  is constant, the velocity of block *B* in 2 s is thus

$$(+\downarrow)$$

$$\begin{aligned} v &= v_0 + a_B t \\ &= 0 + (-6.54)(2) \\ &= -13.1 \text{ m/s} \end{aligned}$$

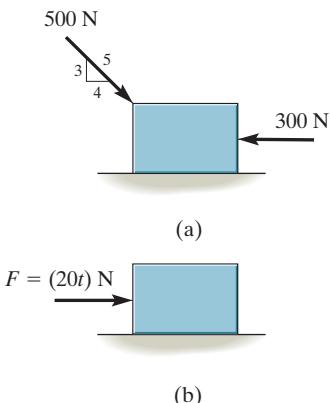
*Ans.*

The negative sign indicates that block *B* is moving upward.

Fig. 13–10

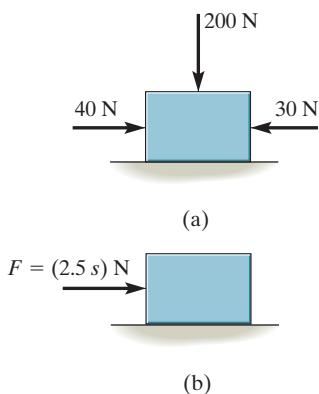
## PRELIMINARY PROBLEMS

**P13-1.** The 10-kg block is subjected to the forces shown. In each case, determine its velocity when  $t = 2$  s if  $v = 0$  when  $t = 0$ .



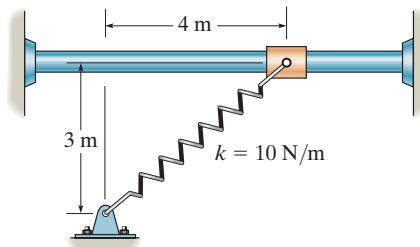
**Prob. P13-1**

**P13-2.** The 10-kg block is subjected to the forces shown. In each case, determine its velocity at  $s = 8$  m if  $v = 3$  m/s at  $s = 0$ . Motion occurs to the right.



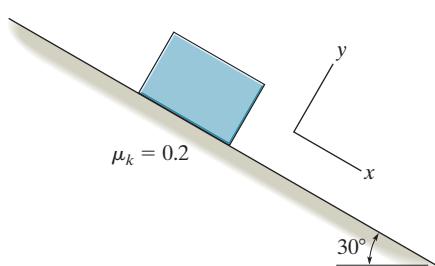
**Prob. P13-2**

**P13-3.** Determine the initial acceleration of the 10-kg smooth collar. The spring has an unstretched length of 1 m.



**Prob. P13-3**

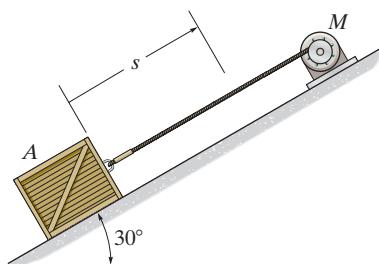
**P13-4.** Write the equations of motion in the  $x$  and  $y$  directions for the 10-kg block.



**Prob. P13-4**

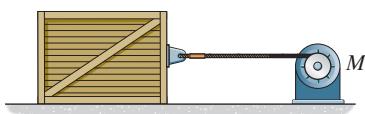
## FUNDAMENTAL PROBLEMS

**F13-1.** The motor winds in the cable with a constant acceleration, such that the 20-kg crate moves a distance  $s = 6 \text{ m}$  in 3 s, starting from rest. Determine the tension developed in the cable. The coefficient of kinetic friction between the crate and the plane is  $\mu_k = 0.3$ .



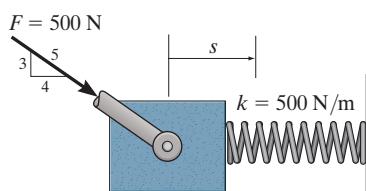
Prob. F13-1

**F13-2.** If motor  $M$  exerts a force of  $F = (10t^2 + 100) \text{ N}$  on the cable, where  $t$  is in seconds, determine the velocity of the 25-kg crate when  $t = 4 \text{ s}$ . The coefficients of static and kinetic friction between the crate and the plane are  $\mu_s = 0.3$  and  $\mu_k = 0.25$ , respectively. The crate is initially at rest.



Prob. F13-2

**F13-3.** A spring of stiffness  $k = 500 \text{ N/m}$  is mounted against the 10-kg block. If the block is subjected to the force of  $F = 500 \text{ N}$ , determine its velocity at  $s = 0.5 \text{ m}$ . When  $s = 0$ , the block is at rest and the spring is uncompressed. The contact surface is smooth.



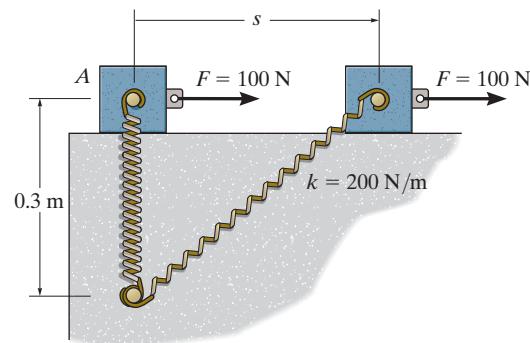
Prob. F13-3

**F13-4.** The 2-Mg car is being towed by a winch. If the winch exerts a force of  $T = 100(s + 1) \text{ N}$  on the cable, where  $s$  is the displacement of the car in meters, determine the speed of the car when  $s = 10 \text{ m}$ , starting from rest. Neglect rolling resistance of the car.



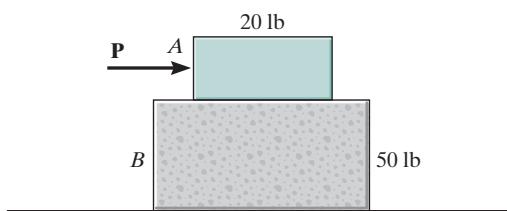
Prob. F13-4

**F13-5.** The spring has a stiffness  $k = 200 \text{ N/m}$  and is unstretched when the 25-kg block is at  $A$ . Determine the acceleration of the block when  $s = 0.4 \text{ m}$ . The contact surface between the block and the plane is smooth.



Prob. F13-5

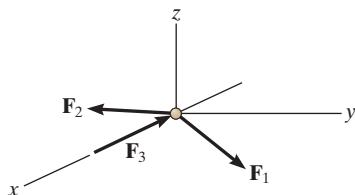
**F13-6.** Block  $B$  rests upon a smooth surface. If the coefficients of static and kinetic friction between  $A$  and  $B$  are  $\mu_s = 0.4$  and  $\mu_k = 0.3$ , respectively, determine the acceleration of each block if  $P = 6 \text{ lb}$ .



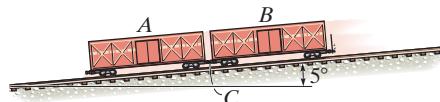
Prob. F13-6

## PROBLEMS

**13-1.** The 6-lb particle is subjected to the action of its weight and forces  $\mathbf{F}_1 = \{2\mathbf{i} + 6\mathbf{j} - 2\mathbf{k}\}$  lb,  $\mathbf{F}_2 = \{t^2\mathbf{i} - 4\mathbf{j} - 1\mathbf{k}\}$  lb, and  $\mathbf{F}_3 = \{-2\mathbf{i}\}$  lb, where  $t$  is in seconds. Determine the distance the ball is from the origin 2 s after being released from rest.

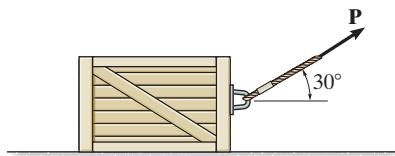
**Prob. 13-1**

**13-2.** The two boxcars *A* and *B* have a weight of 20 000 lb and 30 000 lb, respectively. If they are freely coasting down the incline when the brakes are applied to all the wheels of car *A*, determine the force in the coupling *C* between the two cars. The coefficient of kinetic friction between the wheels of *A* and the tracks is  $\mu_k = 0.5$ . The wheels of car *B* are free to roll. Neglect their mass in the calculation. *Suggestion:* Solve the problem by representing single resultant normal forces acting on *A* and *B*, respectively.

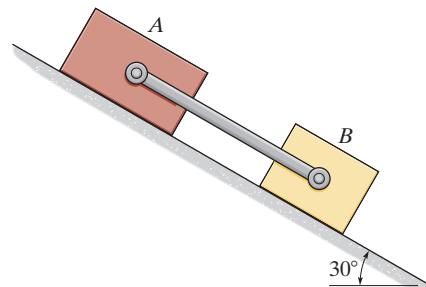
**Prob. 13-2**

**13-3.** If the coefficient of kinetic friction between the 50-kg crate and the ground is  $\mu_k = 0.3$ , determine the distance the crate travels and its velocity when  $t = 3$  s. The crate starts from rest, and  $P = 200$  N.

**\*13-4.** If the 50-kg crate starts from rest and achieves a velocity of  $v = 4$  m/s when it travels a distance of 5 m to the right, determine the magnitude of force  $\mathbf{P}$  acting on the crate. The coefficient of kinetic friction between the crate and the ground is  $\mu_k = 0.3$ .

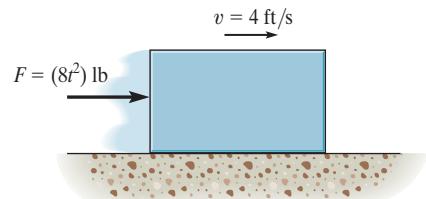
**Probs. 13-3/4**

**13-5.** If blocks *A* and *B* of mass 10 kg and 6 kg respectively, are placed on the inclined plane and released, determine the force developed in the link. The coefficients of kinetic friction between the blocks and the inclined plane are  $\mu_A = 0.1$  and  $\mu_B = 0.3$ . Neglect the mass of the link.

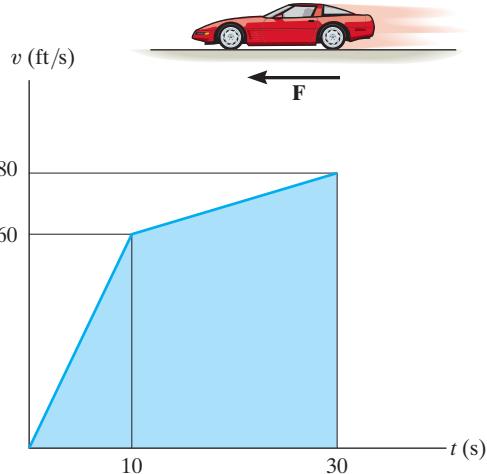
**Prob. 13-5**

**13-6.** The 10-lb block has a speed of 4 ft/s when the force of  $F = (8t^2)$  lb is applied. Determine the velocity of the block when  $t = 2$  s. The coefficient of kinetic friction at the surface is  $\mu_k = 0.2$ .

**13-7.** The 10-lb block has a speed of 4 ft/s when the force of  $F = (8t^2)$  lb is applied. Determine the velocity of the block when it moves  $s = 30$  ft. The coefficient of kinetic friction at the surface is  $\mu_s = 0.2$ .

**Probs. 13-6/7**

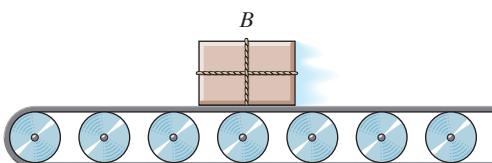
**\*13-8.** The speed of the 3500-lb sports car is plotted over the 30-s time period. Plot the variation of the traction force  $\mathbf{F}$  needed to cause the motion.



Prob. 13-8

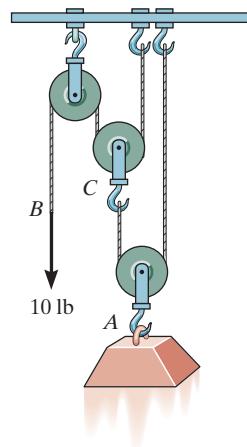
**13-9.** The conveyor belt is moving at 4 m/s. If the coefficient of static friction between the conveyor and the 10-kg package  $B$  is  $\mu_s = 0.2$ , determine the shortest time the belt can stop so that the package does not slide on the belt.

**13-10.** The conveyor belt is designed to transport packages of various weights. Each 10-kg package has a coefficient of kinetic friction  $\mu_k = 0.15$ . If the speed of the conveyor is 5 m/s, and then it suddenly stops, determine the distance the package will slide on the belt before coming to rest.



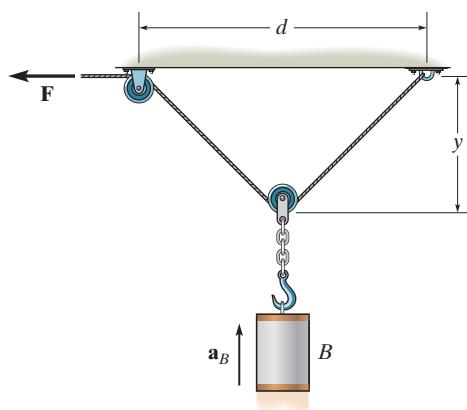
Probs. 13-9/10

**13-11.** Determine the time needed to pull the cord at  $B$  down 4 ft starting from rest when a force of 10 lb is applied to the cord. Block  $A$  weighs 20 lb. Neglect the mass of the pulleys and cords.



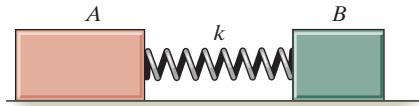
Prob. 13-11

**\*13-12.** Cylinder  $B$  has a mass  $m$  and is hoisted using the cord and pulley system shown. Determine the magnitude of force  $\mathbf{F}$  as a function of the block's vertical position  $y$  so that when  $\mathbf{F}$  is applied the block rises with a constant acceleration  $\mathbf{a}_B$ . Neglect the mass of the cord and pulleys.



Prob. 13-12

- 13-13.** Block A has a weight of 8 lb and block B has a weight of 6 lb. They rest on a surface for which the coefficient of kinetic friction is  $\mu_k = 0.2$ . If the spring has a stiffness of  $k = 20 \text{ lb/ft}$ , and it is compressed 0.2 ft, determine the acceleration of each block just after they are released.



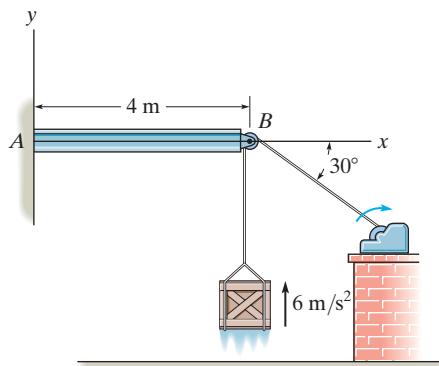
Prob. 13-13

- 13-14.** The 2-Mg truck is traveling at 15 m/s when the brakes on all its wheels are applied, causing it to skid for a distance of 10 m before coming to rest. Determine the constant horizontal force developed in the coupling C, and the frictional force developed between the tires of the truck and the road during this time. The total mass of the boat and trailer is 1 Mg.



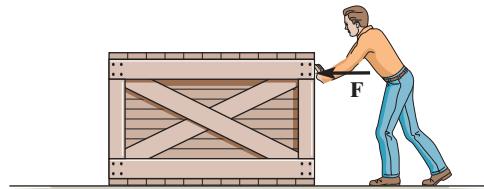
Prob. 13-14

- 13-15.** The motor lifts the 50-kg crate with an acceleration of  $6 \text{ m/s}^2$ . Determine the components of force reaction and the couple moment at the fixed support A.



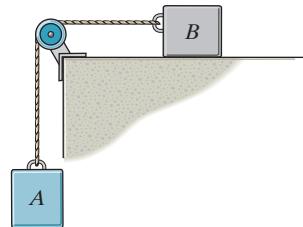
Prob. 13-15

- \*13-16.** The 75-kg man pushes on the 150-kg crate with a horizontal force  $F$ . If the coefficients of static and kinetic friction between the crate and the surface are  $\mu_s = 0.3$  and  $\mu_k = 0.2$ , and the coefficient of static friction between the man's shoes and the surface is  $\mu_s = 0.8$ , show that the man is able to move the crate. What is the greatest acceleration the man can give the crate?



Prob. 13-16

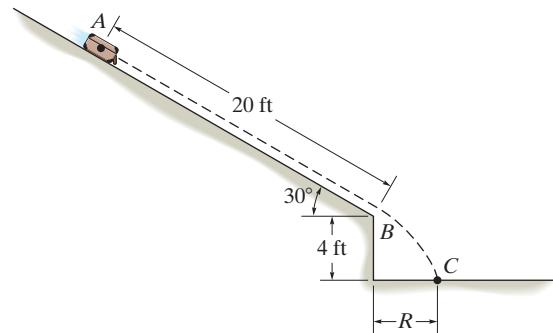
- 13-17.** Determine the acceleration of the blocks when the system is released. The coefficient of kinetic friction is  $\mu_k$ , and the mass of each block is m. Neglect the mass of the pulleys and cord.



Prob. 13-17

- 13-18.** A 40-lb suitcase slides from rest 20 ft down the smooth ramp. Determine the point where it strikes the ground at C. How long does it take to go from A to C?

- 13-19.** Solve Prob. 13-18 if the suitcase has an initial velocity down the ramp of  $v_A = 10 \text{ ft/s}$  and the coefficient of kinetic friction along AB is  $\mu_k = 0.2$ .



Probs. 13-18/19