# TTK4190 Guidance and Control of Vehicles

# Assignment 3 - Part 1: Marine Craft Modeling

October 14, 2020 Group 02

# 1 Problem 1: Rigid-Body Kinetics of a Rectangular Prism

## 1.a Moments of Intertia about CG

$$\rho_m = \frac{m}{LBH} = \frac{17.0677 \times 10^6}{161 \times 21.8 \times 15.8} = 307.78 \tag{1}$$

Now we can calculate the moments of inertia:

$$\boldsymbol{I}_{z}^{CG} = \int_{V} (x^2 + y^2) \rho_m \, \mathrm{d}V \tag{2}$$

$$= \rho_m H \int_{-B/2}^{B/2} \int_{-L/2}^{L/2} (x^2 + y^2) \, \mathrm{d}x \, \mathrm{d}y$$
 (3)

$$=3.7544 \times 10^{10} \tag{4}$$

With similar formulas it's easy to see why all the cross term inertias are 0:

$$\boldsymbol{I}_{yx}^{CG} = \boldsymbol{I}_{xy}^{CG} = \int_{V} xy \rho_m \, dV \tag{5}$$

$$= \rho_m H \int_{-B/2}^{B/2} \int_{-L/2}^{L/2} xy \, dx \, dy \tag{6}$$

$$=0 (7)$$

Intuitively this also makes sense as the "ship" is symmetric about CG.

# 1.b Moment of Inertia about CO

With the distance vector given by

$$\boldsymbol{r}_{bg}^b = \begin{bmatrix} -3.7 & 0 & H/2 \end{bmatrix}^\mathsf{T}$$

the moment of inertia about the z in CO is given by:

$$I_z^{CO} = I_z^{CG} + m(x^2 + y^2)$$
(8)

$$= 3.7544 \times 10^{10} + 17.0677 \times 10^{6} \times (-3.7)^{2}$$
(9)

$$=3.7778 \times 10^{10} \tag{10}$$

## 1.c Mass and Coriolis matrices

The analytical expressions for the rigid body mass,  $M_{RB}$ , and the rigid body Coriolis,  $C_{RB}$  for the three DOGs surge, sway and yaw, using  $r_{bg}^b$  from task b, were found to be

$$\mathbf{M}_{RB} = \begin{bmatrix} m & 0 & -my_g \\ 0 & m & mx_g \\ -my_g & mx_g & I_z \end{bmatrix}$$
 (11)

$$C_{RB} = \begin{bmatrix} 0 & 0 & -m(x_g r + v) \\ 0 & 0 & -m(y_g r - u) \\ m(x_g r + v) & m(y_g r - u) & 0 \end{bmatrix},$$
(12)

where u and v describes the linear velocity of the CO relative to  $o_n$  expressed in the body frame.

## 1.d Skew-Symmetric Coriolis

We see that  $C_{RB} = C_{RB}^T$ , which implies that the rigid body Coriolis matrix is skew symmetric. This is desired because the quadratic form  $\nu^T C_{RB}(\nu)\nu = 0$ , which is exploited in energy based designs where the Lyaounov function is important.

# 1.e Other property of Coriolis matrix

The expression for  $C_{RB}(\nu)$  can be made independent of linear velocity  $v_1$ . This is done by using the cross-product property  $S(v_1)v_2 = -S(v_2)v_1$ , and move  $S(v_1)v_2$  from  $C_{RB}^{12}$  to  $C_{RB}^{11}$ . This property becomes useful when irrational ocean currents enter the EoM, as  $C_{RB}(\nu)$  is independent on linear velocity  $v_1$  and only uses the angular velocity  $v_2$  and lever arm  $r_{bq}^b$ .

# 2 Problem 2: Hydrostatics

#### 2.a Volume displacement

With the constant draft, we assume the ship is neutrally buoyant so the volume displacement can be calculated with Archimedes' principle:

$$mg = \rho g \nabla \tag{13}$$

$$\implies \nabla = \rho/m \tag{14}$$

Seeing as our ship model is a prism an easier way of calculating the volume displacement would simply be:

$$\nabla = LBT = 1.66 \times 10^4 \tag{15}$$

#### 2.b Waterplane and hydrostatic forces

From (4.14) in Fossen we have:

$$Z_{HS} \approx -\rho g A_{wp} z^n \tag{16}$$

$$= -\rho g L B T = -3.14 \times 10^8 \tag{17}$$

#### 2.c Heave period

$$T_3 \approx 2\pi \sqrt{\frac{2T}{g}} = 8.46 \tag{18}$$

# 2.d Metacentric heights

The metacentric heights are given by:

$$GM_T = \frac{I_T}{\nabla} + z_g - z_b, \quad GM_L = \frac{I_L}{\nabla} + z_g - z_b$$
 (19)

Where the intertias are given by:

$$I_T = \frac{1}{A_{wp}} \int \int_{A_{wp}} y^2 \, \mathrm{d}A = L \int_{-B/2}^{B/2} y^2 \, \mathrm{d}y = \frac{1}{12} B L^3$$
 (20)

$$I_L = \frac{1}{12} B^3 L (21)$$

With the center of bouyancy in T/2 we have that  $z_g - z_b = T/2 - H/2$ . Inserting this in equation 19 together with equation 15 we get:

$$GM_T = \frac{L^2}{12T} + \frac{T}{2} - \frac{H}{2} = 0.9998 \tag{22}$$

$$GM_L = \frac{B^2}{12T} + \frac{T}{2} - \frac{H}{2} = 239.26 \tag{23}$$

#### 2.e Metacentric stability

By looking at the numerical values for  $GM_T$  and  $GM_L$  in d) we see that the vessel is more metacentrically stable in the longitudinal plane than the transverse plane. The value for  $GM_L$  implies stiffness in this plane, which makes logical sense considering the shape of the vessel.  $GM_T$  on the other hand is much lower, implying less metacentric stability, however as it is not negative or excessively low there is not a risk of the vessel capsizing in rough weather.

# 3 Problem 3: Added Mass and Coriolis

#### 3.a Reduced added mass

With the assumption of no coupling between subsystems the added mass matrix for surge, sway and yaw will simply be:

$$\boldsymbol{M}_{A} = \boldsymbol{M}_{A}^{\mathsf{T}} = -\begin{bmatrix} X_{\dot{u}} & X_{\dot{v}} & X_{\dot{r}} \\ Y_{\dot{u}} & Y_{\dot{v}} & Y_{\dot{r}} \\ N_{\dot{u}} & N_{\dot{v}} & N_{\dot{r}} \end{bmatrix} = -\begin{bmatrix} X_{\dot{u}} & 0 & 0 \\ 0 & Y_{\dot{v}} & Y_{\dot{r}} \\ 0 & N_{\dot{v}} & N_{\dot{r}} \end{bmatrix}$$
(24)

# 3.b Coriolis forces

From Property 6.2 in Fossen we have the full Coriolis matrix given by:

$$C(\nu) = \begin{bmatrix} \mathbf{0}_{3\times3} & -S(A_{11}\nu_1 + A_{12}\nu_2) \\ -S(A_{11}\nu_1 + A_{12}\nu_2) & -S(A_{21}\nu_1 + A_{22}\nu_2) \end{bmatrix}$$
(25)

Picking out the components corresponding to surge, sway and yaw we get:

$$C(\nu) = \begin{bmatrix} 0 & 0 & a_2 \\ 0 & 0 & -a_1 \\ -a_2 & a_1 & 0 \end{bmatrix}$$
 (26)

Where  $a_1$  and  $a_2$  are given by:

$$a_1 = X_{\dot{u}}u + X_{\dot{v}}v + X_{\dot{r}}r = X_{\dot{u}}u \tag{27}$$

$$a_2 = Y_{\dot{u}}u + Y_{\dot{v}}v + Y_{\dot{r}}r = Y_{\dot{v}}v + Y_{\dot{r}}r \tag{28}$$

# 4 Problem 4: Implementing the Maneuvering Model

- 4.a Mass and Coriolis matrices
- 4.b Diagonal linear damping
- 4.c Simulation Plots