

Multibody System Dynamics: MBDyn Primer



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Outline



- Software
- Input File
- Results
- Example 1: Rigid Body
- Example 2: Pendulum
- Example 3: Cantilever Beam
- Example 4: Plane Mechanism

Software



- Web site: http://www.mbdyn.org/
- Distributed in source form
- Developed for Linux
- Needs Un*x-like build environment
- Binaries compiled for Windows XP available here: http://www.aero.polimi.it/masarati/Download/mbdyn/
- Latest: http://www.aero.polimi.it/masarati/Download/mbdyn/mbdyn-1.5.6-win32.zip

Software



- Command-line software
- Prepare an input file using your favourite text editor
- execute:

```
# mbdyn.exe input_file -o output_file
```

- Output in files with specific extensions (discussed later)
- Load output files in math environment (octave, scilab, matlab, ...)
 and use results (e.g. for plotting)

Software



- Output can be reformatted for some post-processing tools
 - EasyAnim
 - ...
- Ongoing third-party project about using Blender http://www.blender.org/ for pre/post-processing
- Output also available in binary form for NetCDF http://www.unidata.ucar.edu/software/netcdf/



- The model and the analysis are defined in an input file
- Use your preferred editor to prepare the input file
- The structure and the syntax of the statements are described here https://www.mbdyn.org/userfiles/documents/mbdyn-input-1.5.6.pdf (pick the manual for the version in use)
- A set of tutorials is presented here https://www.mbdyn.org/userfiles/documents/tutorials.pdf



The input is organized in statements:

```
<statement> [ : <arglist> ] ;
```

- Statements are terminated by a semicolon.
- If no args expected:

```
<statement> ;
```

- Arguments are comma-separated
- Keywords are case-insensitive; data is case-sensitive
- When a numerical value is expected, a mathematical parser is called to evaluate math expressions; example:

```
structural nodes:
+1  # first node
+10*2  # other 20 nodes
;  # i.e. 21 structural nodes will be defined
```



File structure: blocks of statements

```
# data block (type of analysis)
# <type> block (analysis parameters)
# control data block (model-specific general data)
# nodes block
# (optional) drivers block
# elements block
```



Data block

```
begin: data;
    problem: initial value;
end: data;
```

We only consider initial value problems by now.



"Initial value" block: define the parameters of the analysis

```
begin: initial value;
   time step: real DT = 1e-3;
   initial time: real INITIAL_TIME = 0.;
   final time: INITIAL_TIME + 1000*DT;

   tolerance: 1e-6;
   max iterations: 10;

# linear solver: naive, colamd, mt, 1;
   # nonlinear solver: newton raphson, modified, 5;
   # method: ms, 0.6;
end: initial value;
```

Other statements can be required



"Initial value" block: define the parameters of the analysis

```
# Better style:
begin: initial value;
    set: real DT = 1e-3;
    set: real INITIAL_TIME = 0.;
    time step: DT;
    initial time: INITIAL_TIME;
    final time: INITIAL TIME + 1000*DT;
    tolerance: 1e-6;
    max iterations: 10;
    # linear solver: naive, colamd, mt, 1;
    # nonlinear solver: newton raphson, modified, 5;
    # method: ms, 0.6;
end: initial value;
```



Control data block: define the parameters of the model

```
begin: control data;
    structural nodes: 1;
    rigid bodies: 1;
    gravity;
end: control data;
```

- Other entities and parameters can be declared/defined
- Rationale:
 - Declare the count of each entity type expected later
 - Define parameters related to model initialization
 - Define parameters related to model handling (output...)



- Miscellaneous statements (can appear anywhere)
- The "set" statements calls the math parser:

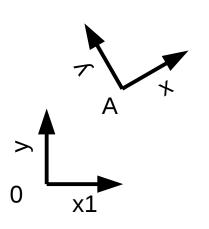
```
set: real X = 0.;
```

The "reference" statement defines a reference frame:

```
set: integer LABEL = 1000;
reference: LABEL,
    reference, global, 0., 0., # position
    reference, global, # orientation
        1, 1., 0., 0.,
        3, 0., 0., 1.,
    reference, global, 0., 0., # velocity
    reference, global, 0., 0., 0.; # angular velocity
```



- References can be defined incrementally:
 a new reference can refer to an already defined one
- The combined use of symbolic names and data, and of hierarchical references, allows to build parametric models



```
set: integer POINT_0 = 1;
set: integer POINT_A = 2;
set: real OMEGA = 99.9;
set: real ALPHA = 30.0*deg2rad;
reference: POINT_0,
    reference, global, null,
    reference, global, eye,
    reference, global, null,
    reference, global, 0.0, 0.0, OMEGA;
reference: POINT_A,
    reference, POINT_0, 10.0, 20.0, 0.0,
    reference, POINT_0,
        3, 0.0, 0.0, 1.0,
        1, cos(ALPHA), sin(ALPHA), 0.0,
    reference, POINT_0, null,
    reference, POINT_0, null;
```



- References do not participate in the simulation
- References are only used during model input
- References inherit:
 - position,
 - orientation,
 - velocity
 - angular velocity

of the references they refer to:

$$\mathbf{x}_{1} = \mathbf{x}_{0} + \mathbf{R}_{0} \mathbf{x}'$$

$$\mathbf{R}_{1} = \mathbf{R}_{0} \mathbf{R}'$$

$$\mathbf{v}_{1} = \mathbf{v}_{0} + \boldsymbol{\omega}_{0} \times \mathbf{R}_{0} \mathbf{x}' + \mathbf{R}_{0} \mathbf{v}'$$

$$\boldsymbol{\omega}_{1} = \boldsymbol{\omega}_{0} + \mathbf{R}_{0} \boldsymbol{\omega}'$$



Nodes:

```
begin: nodes;
    set: integer NODE_1 = 100;
    # a node with initial velocity in x
    structural: NODE_1, dynamic,
        reference, global, null,
        reference, global, eye,
        reference, global, null,
        reference, global, 10., 0.;
end: nodes;
```



- Nodes instantiate kinematic degrees of freedom and the corresponding equilibrium equations
- Static nodes only instantiate equilibrium equations

$$0 = \sum_{\mathbf{m}} \mathbf{f}$$
$$0 = \sum_{\mathbf{m}} \mathbf{m}$$

Dynamic nodes also instantiate momentum and momenta moment definitions

$$\mathbf{M} \dot{\mathbf{x}} = \beta$$

$$\mathbf{J} \boldsymbol{\omega} = \gamma$$

$$\dot{\boldsymbol{\beta}} = \sum_{\mathbf{f}} \mathbf{f}$$

$$\dot{\boldsymbol{\gamma}} = \sum_{\mathbf{m}} \mathbf{m}$$

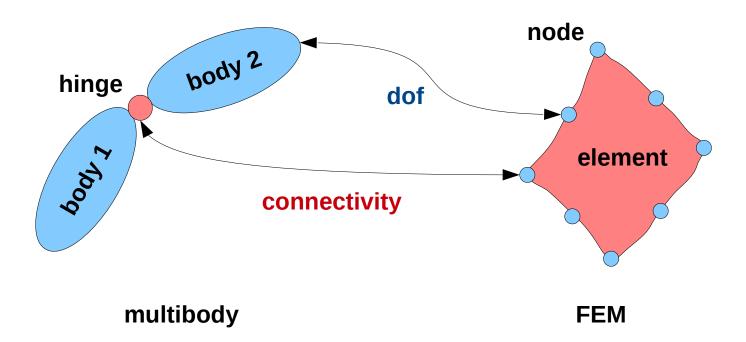


Elements:

- Elements write contributions to nodes equations
- Elements represent "connectivity" and "constitutive properties"
- Elements can add further, "private" equations (e.g. algebraic constraints)



 Multibody vs. FEM: nodes at bodies vs. nodes at frontier (node → body; element → hinge)



Output



outfile.mov: structural node motion

- One row for each node
- One block for each time step
- Columns:
 - #1: node label
 - #2 4: position components
 - #5 7: orientation parameters
 - #8 10: velocity components
 - #11 13: angular velocity components
- Acceleration and angular acceleration can be requested (only for dynamic nodes)
- All data is in the global reference frame (exceptions on demand)

Output



outfile.jnt: joints output

- One row for each joint
- One block for each time step
- Columns: each joint type differs
- Column number may differ (cannot be loaded in matlab)
- Joint output documented in input manual



- Other node and element types have their output file and format (presented and discussed in tutorials and input manual)
- Experimental output on database (NetCDF)
- Only structural nodes and little more currently implemented
- More efficient:
 - Less disk space
 - No overhead in formatting output
 - Higher precision (more digits)
- Octave, matlab, many other software interpret the format

Rigid Body



- Model a body that falls freely, subjected to gravity
- Hints:
- Use a dynamic structural node
- Connect a rigid body to it
- Apply gravity



- Model a pendulum, subjected to gravity
- Hints:
- Use two structural nodes: ground and pendulum
- Use a "clamp" joint to ground the ground node
- Use a "revolute hinge" joint to ground the pendulum node
- Connect a rigid body to the pendulum node
- Add gravity

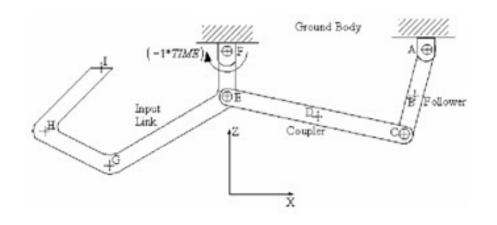
Cantilever Beam



- Model a cantilever beam subjected to a load at the free end, using a variable number of 3 node beam elements
- Hints:
- Ground one structural node
- Add two more nodes for each beam
- Add a rigid body for each non-grounded node



- Model a plane mechanism, consisting in three bodies hinged together and to the ground, according to the figure
- Add a spring about hinge in point A
- Load the system with a force in point I



point	Х	Z
Α	0.921	1.124
В	0.918	1.114
С	0.915	1.104
D	0.896	1.106
E	0.878	1.108
F	0.878	1.118
G	0.830	1.080
Н	0.790	1.088
l	0.825	1.109



