

# trajectory\_utils: Mathematical background

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## 1. Cartpoles

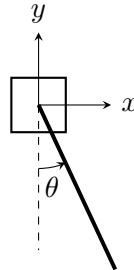


Figure 1: Cart–pole system. The cart is actuated by some means and is constrained to move along  $\pm x$ . Objectives are to swing the pole up to vertical - and/or stabilize it there. The physical cart is actuated by a belt drive, and the pole is a steel rod.

### 1.1 Equations of motion

By inspection, figure 1 offers:

$$x_r(t) = r \sin \theta(t), r \in [0, L] \quad (1)$$

$$y_r(t) = -r \cos \theta(t), r \in [0, L] \quad (2)$$

$$\dot{x}_r(t) = r \cos \theta(t) \dot{\theta}(t) + \dot{x}(t), r \in [0, L] \quad (3)$$

$$\dot{y}_r(t) = r \sin \theta(t) \dot{\theta}(t), r \in [0, L] \quad (4)$$

where  $r$  parameterizes the location of infinitesimal mass along the pole. It is straightforward to compute the Lagrangian  $\mathcal{L} = T - V$  - the first step in deriving the equations of motion.

The kinetic energy  $T$  can be written as a sum of the cart kinetic energy and an integral over the pole:

$$T = \int_{r=0}^{r=L} (\dot{x}_r^2 + \dot{y}_r^2) \rho dr + \frac{1}{2} m_c \dot{x}^2 \quad (5)$$

where  $\rho$  is the pole mass per unit length. A bit of algebra yields:

$$T = m_p \frac{L^2}{6} \dot{\theta}^2 + \frac{1}{2} m_p L \cos \theta \dot{\theta} \dot{x} + \frac{1}{2} (m_p + m_c) \dot{x}^2 \quad (6)$$

where we have used the fact that  $\rho L = m_p$ .

The (gravitational) potential energy of the cart does not change, so we are left with an integral over the pole:

$$V = g \int_{r=0}^{r=L} y_r \rho dr \quad (7)$$

or, equivalently, the potential energy of a mass concentrated at the pole's center of mass:

$$V = -\frac{gm_p L}{2} \cos \theta \quad (8)$$

Euler-Lagrange now yields two equations:

$$\frac{d}{dt} \left( \frac{\partial \mathcal{L}}{\partial \dot{x}} \right) - \frac{\partial \mathcal{L}}{\partial x} = F_x \quad (9)$$

$$\frac{d}{dt} \left( \frac{\partial \mathcal{L}}{\partial \dot{\theta}} \right) - \frac{\partial \mathcal{L}}{\partial \theta} = 0 \quad (10)$$

The first (force) equation yields:

$$\frac{1}{2} m_p L \left( \cos \theta \ddot{\theta} - \sin \theta \dot{\theta}^2 \right) + (m_p + m_c) \ddot{x} = F_x \quad (11)$$

and the second (torque) equation yields:

$$\frac{2}{3} L \ddot{\theta} + \cos \theta \ddot{x} + g \sin \theta = 0 \quad (12)$$

or, conveniently:

$$\ddot{\theta} = -\frac{3}{2L} (\cos \theta \ddot{x} + g \sin \theta) \quad (13)$$

This equation alone can be used to establish an ODE for the cartpole system with the “velocity” servo actuation model:

$$\dot{v} = \frac{v_c - v}{\tau} \quad (14)$$

or:

$$\ddot{x} = \frac{v_c - \dot{x}}{\tau} \quad (15)$$

where  $v_c$  is the velocity control. The state ODE for this system is below.

For cart force actuation, we can decouple the above (coupled) equations in  $\ddot{\theta}$  and  $\ddot{x}$  by thrashing about with some algebra, yielding:

$$\ddot{\theta} = -\frac{6F_x \cos \theta + 6g \sin \theta (m_c + m_p) + 3m_p L \sin \theta \cos \theta \dot{\theta}^2}{dL} \quad (16)$$

$$\ddot{x} = \frac{4F_x \cos \theta + 3gm_p \sin \theta \cos \theta + 2m_p L \sin \theta \dot{\theta}^2}{d} \quad (17)$$

where the denominator factor  $d$  is:

$$d = 4m_c + m_p (1 + 3 \sin^2 \theta) \quad (18)$$

Below we assume the following state space:

$$s = \begin{bmatrix} x \\ \theta \\ \dot{x} \\ \dot{\theta} \end{bmatrix} \quad (19)$$

## 1.2 Cart force control

The above equations yield the following first order cartpole system, assuming force control:

$$\dot{s} = \begin{bmatrix} \dot{x} \\ \dot{\theta} \\ \ddot{x} \\ \ddot{\theta} \end{bmatrix} = \begin{bmatrix} \dot{x} \\ \dot{\theta} \\ \frac{4F_x \cos \theta + 3gm_p \sin \theta \cos \theta + 2m_p L \sin \theta \dot{\theta}^2}{d} \\ -\frac{6F_x \cos \theta + 6g \sin \theta (m_c + m_p) + 3m_p L \sin \theta \cos \theta \dot{\theta}^2}{dL} \end{bmatrix} \quad (20)$$

## 1.3 Cart velocity servo control

For the velocity servo control, we have the simpler (force-free) equations:

$$\dot{s} = \begin{bmatrix} \dot{x} \\ \dot{\theta} \\ \ddot{x} \\ \ddot{\theta} \end{bmatrix} = \begin{bmatrix} \dot{x} \\ \dot{\theta} \\ \frac{v_c - \dot{x}}{\tau} \\ -\frac{3}{2L} \left( \frac{v_c - \dot{x}}{\tau} \cos \theta + g \sin \theta \right) \end{bmatrix} \quad (21)$$

- 1.4 The cvxpy experience**
- 2. Differential Drive Control trajectories**
  - 2.1 Signed Distance Function (SDF) for obstacle avoidance**
  - 2.2 Differentiable SDF in pyTorch**
  - 2.3 The cvxpy experience**
- References**